



# **Reaction wheel based Attitude control system**

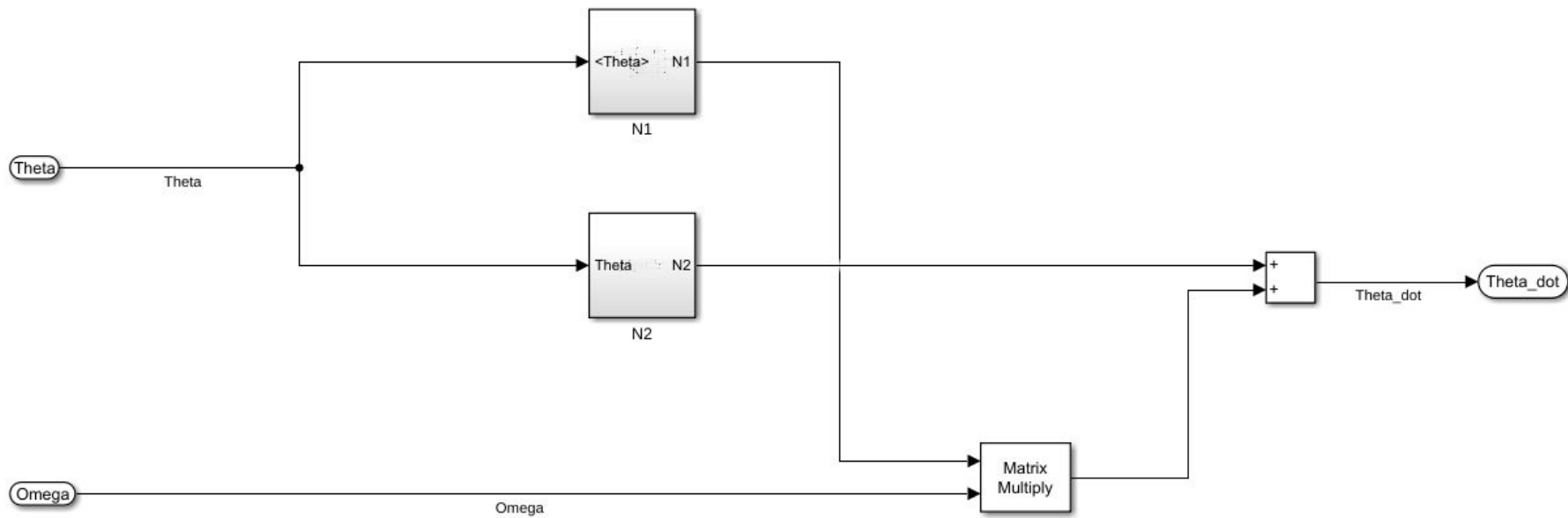
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# Attitude Dynamics

For spacecraft in a circular orbit with an orbital rate  $n = \sqrt{\mu/R^3}$

$$\begin{bmatrix} \dot{\theta}_1 \\ \dot{\theta}_2 \\ \dot{\theta}_3 \end{bmatrix} = \frac{1}{\cos \theta_2} \begin{bmatrix} \cos \theta_2 & \sin \theta_1 \sin \theta_2 & \cos \theta_1 \sin \theta_2 \\ 0 & \cos \theta_1 \cos \theta_2 & -\sin \theta_1 \cos \theta_2 \\ 0 & \sin \theta_1 & \cos \theta_1 \end{bmatrix} \begin{bmatrix} \omega_1 \\ \omega_2 \\ \omega_3 \end{bmatrix} + \frac{n}{\cos \theta_2} \begin{bmatrix} \sin \theta_3 \\ \cos \theta_2 \cos \theta_3 \\ \sin \theta_2 \sin \theta_3 \end{bmatrix}$$

$$\Rightarrow \boxed{\dot{\bar{\theta}} = N_{1E}(\bar{\theta})\omega + nN_{2E}(\bar{\theta})}$$






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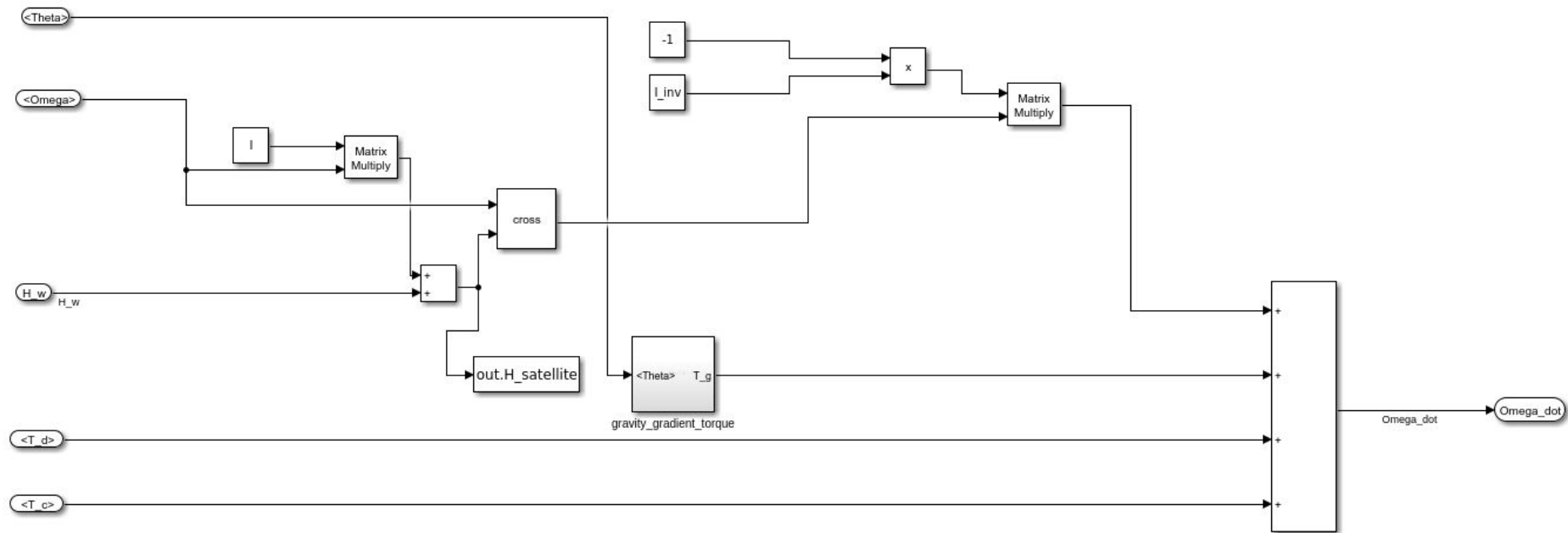

$$J \begin{bmatrix} \dot{\omega}_1 \\ \dot{\omega}_2 \\ \dot{\omega}_3 \end{bmatrix} = - \begin{bmatrix} 0 & -\omega_1 & \omega_2 \\ \omega_3 & 0 & -\omega_1 \\ \omega_2 & \omega_1 & 0 \end{bmatrix} J \begin{bmatrix} \omega_1 \\ \omega_2 \\ \omega_3 \end{bmatrix} + 3n^2 \begin{bmatrix} 0 & -\cos \theta_1 \cos \theta_2 & \sin \theta_1 \cos \theta_2 \\ \cos \theta_1 \cos \theta_2 & 0 & \sin \theta_2 \\ -\sin \theta_1 \cos \theta_2 & -\sin \theta_2 & 0 \end{bmatrix} J \begin{bmatrix} -\sin \theta_2 \\ \sin \theta_1 \cos \theta_2 \\ \cos \theta_1 \cos \theta_2 \end{bmatrix} + \overline{T_d} + \overline{T_c}$$


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$$\Rightarrow \boxed{J\dot{\bar{\omega}} = -S(\bar{\omega})J\bar{\omega} + \overline{T_g} + \overline{T_d} + \overline{T_c}}$$

To also include the contribution of angular momentum of reaction wheels

$$\boxed{J\dot{\bar{\omega}} = -\bar{\omega} \times (J\bar{\omega} + H_{rw}) + \overline{T_g} + \overline{T_d} + \overline{T_c}}$$





# Reaction wheel

4 wheels in Pyramidal configuration.  
 $\beta$  - tilt angle of reaction wheel axes

$$\begin{bmatrix} \hat{T}_{cx} \\ \hat{T}_{cy} \\ \hat{T}_{cz} \end{bmatrix} = \begin{bmatrix} T_{cx} / \cos(\beta) \\ T_{cy} / \cos(\beta) \\ T_{cz} / \sin(\beta) \end{bmatrix} = \begin{bmatrix} 1 & 0 & -1 & 0 \\ 0 & 1 & 0 & -1 \\ 1 & 1 & 1 & 1 \end{bmatrix} \begin{bmatrix} T_1 \\ T_2 \\ T_3 \\ T_4 \end{bmatrix}$$



**For optimal distribution of momentum among the 4 wheels.**

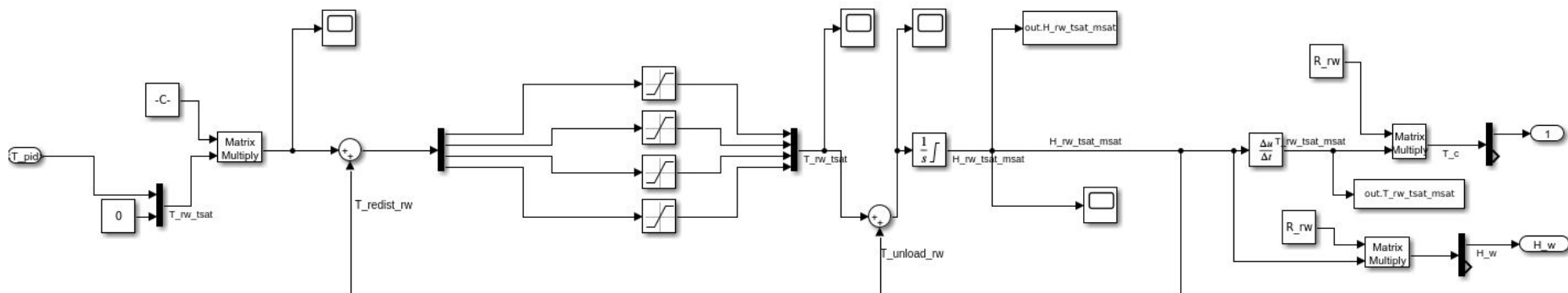
$$\begin{bmatrix} T_1 \\ T_2 \\ T_3 \\ T_4 \end{bmatrix} = \frac{1}{2} \begin{bmatrix} 1 & 0 & 1/2 & 1/2 \\ 0 & 1 & 1/2 & -1/2 \\ -1 & 0 & 1/2 & 1/2 \\ 0 & -1 & 1/2 & -1/2 \end{bmatrix} \begin{bmatrix} \hat{T}_{cx} \\ \hat{T}_{cy} \\ \hat{T}_{cz} \\ 0 \end{bmatrix}$$

Pyramidal reaction wheel arrangement optimization of satellite attitude control subsystem for minimizing power consumption

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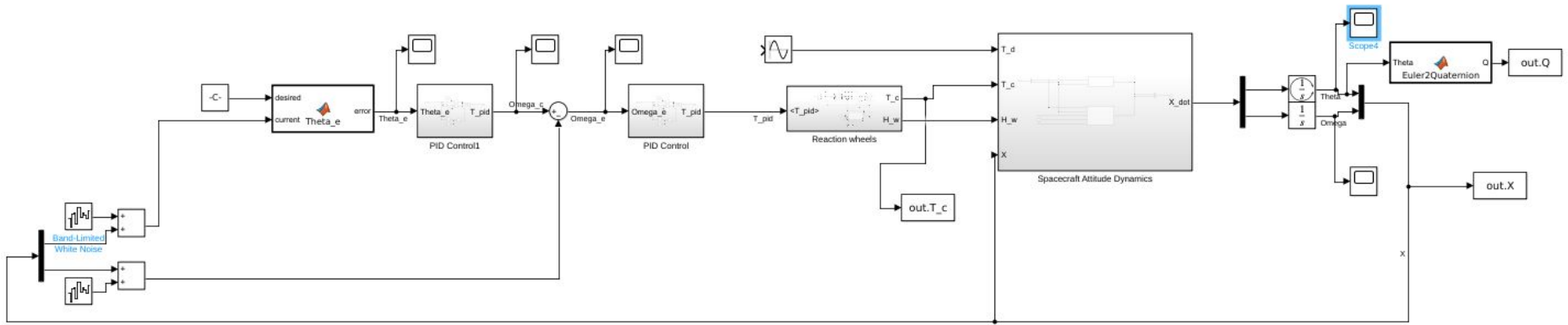




## Cascaded PD control


Outer loop is the primary controller. It converts the error in attitude to a desired angular velocity (Makes it go from initial rates to 0).

The inner loop then takes in the error in angular velocity and commands the reaction wheels. The inner loop must have  $>10$  times faster dynamics



### Available Data

Initial Slew Attitude (deg)	[0 0 0]
Final Slew Attitude (deg)	[30 50 70]
Initial Rate (deg/sec)	[15 -15 15]
Desired Rate (deg/sec)	[0.2 0.2 0.2]
MOI of the Reaction Wheel (kg m <sup>2</sup> )	0.00042
MOI of the satellite (kg m <sup>2</sup> )	[2.10 0.00 0.01; 0.00 2.30 -0.03; 0.01 -0.03 1.72]
Disturbance Torque (Nm) acting on the satellite (sinusoidal in ECI Frame)	[10 <sup>-6</sup> 10 <sup>-6</sup> 10 <sup>-6</sup> ]
Maximum Torque generated (Nm) by the Reaction Wheel	0.015
Angular Momentum generated (Nms) by the Reaction wheel	0.035
Reaction Wheel Configuration	Pyramidal Configuration



For the given initial and final conditions, and the restrictions on maximum reaction wheel torques and angular momentum, the reaction wheels saturate very quickly before the desired final conditions are reached.

Hence, a momentum management system was designed to redistribute the momentum among the 4 wheels and also provide an external unloading torque when necessary.



# Momentum Management System

- Redistribution
- Unloading



## Redistribution

$R_{rw}$  is the matrix that converts 4 individual reaction wheel torques into the body torque

$$\begin{bmatrix} T_b \\ 0 \end{bmatrix} = R_{rw} T_{rw}$$

Redistribution is triggered when any of the reaction wheels exceeds 90% of the max momentum capacity

Let's assume wheel 1 reached  $>0.9 H_{c\_max}$

Then we would like to set the desired momentum of wheel 1 to 0, and distribute that momentum to the other wheels.

Momentum to be redistributed to other wheels,  $\Delta H = [H_1; 0; 0; 0]$

Momentum to be redistributed in 3 axis,  $H_{redist} = R_{rw} \Delta H$

Now we want to know how to distribute the momentum. For that, we make first column of  $R_{rw}$  as 0 (let's call it  $R_{rw\_redist}$ ), so that no momentum will be added to the saturated wheel

$H_{redist\_rw} = \text{inv}(R_{rw\_redist}) * H_{redist}$

Hence, the new desired momentum for the wheels become

$H_{des\_rw} = [0; h_1; h_2; h_3] + H_{redist\_rw}$

$H_{des} = (R_{rw} H_{des\_rw})[1:3]$

The redistribution is performed on the model by calculating a redistribution torque, and then subjecting it to the saturation limits of the reaction wheels.

$T_{redist} = -K_{redist} * (H - H_{des})$

$T_{redist\_rw} = \text{inv}(R_{rw})[T_{redist}; 0]$



## Momentum unloading

It is triggered when sum of momentum of all reaction wheels are  $>0.9 \cdot 4 \cdot H_{c\_max}$

Once triggered, it continues until the sum of momentums of all reaction wheels are  $<0.1 \cdot 4 \cdot H_{c\_max}$ .

Hence, the  $H_{desired\_rw} = 0.1 \cdot H_{c\_max} [1;1;1;1]$ ,

$$H_{des} = R_{rw} \cdot H_{desired\_rw}$$

$$T_{unload} = -K_{unload} \cdot (H - H_{des})$$

$$T_{unload\_rw} = \text{inv}(R_{rw}) [T_{unload}; 0]$$

Assumption - unloading torque is provided by an external source. There is no restriction in the magnitude.





## Simulink model results

1000 seconds

With sinusoidal disturbance torque of  $[10^{-6} \ 10^{-6} \ 10^{-6}]$  Nm but no sensor noise.

Momentum management system gains

$K_{\text{redist}} = [1; 1; 5]$

$K_{\text{unload}} = 1.5 \times 10^{-1}$

PID gains

Outer loop

$K_{p_t} = 10^{-2};$

$K_{d_t} = 10^{-4};$

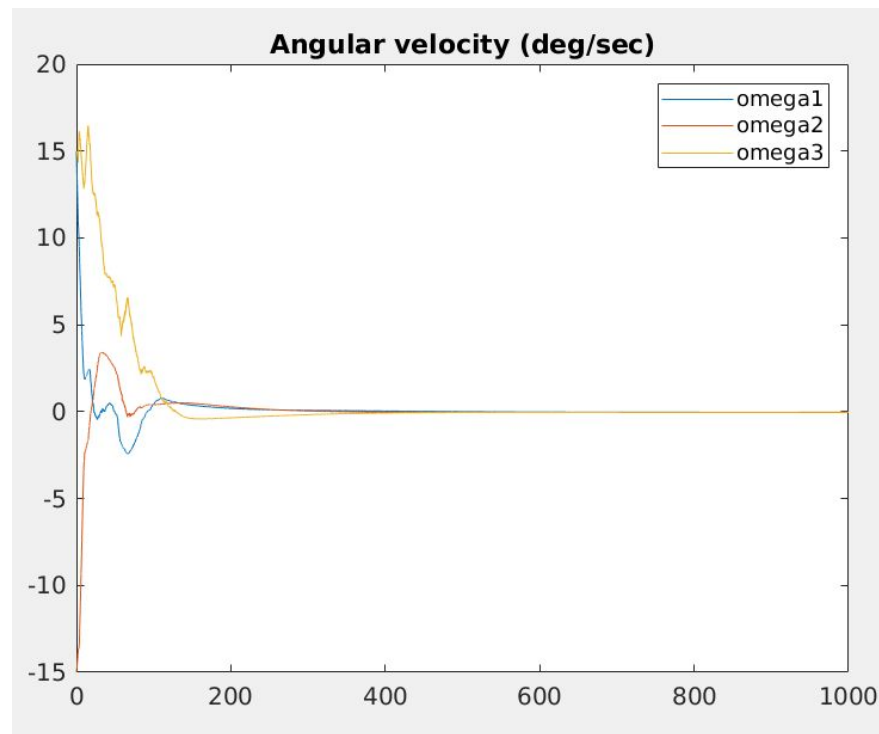
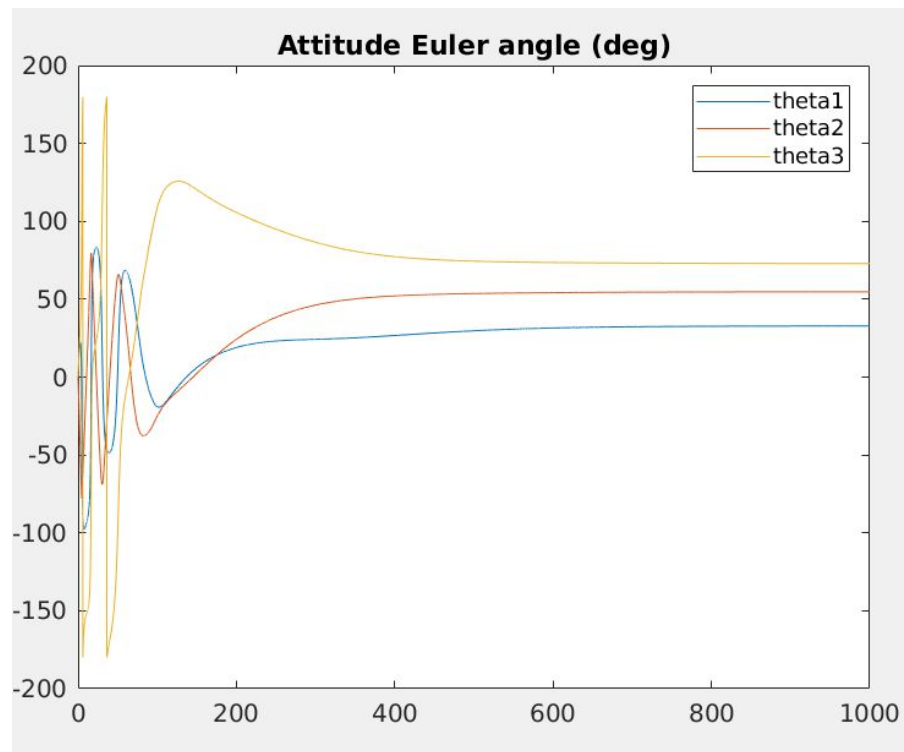
$K_{i_t} = 0;$

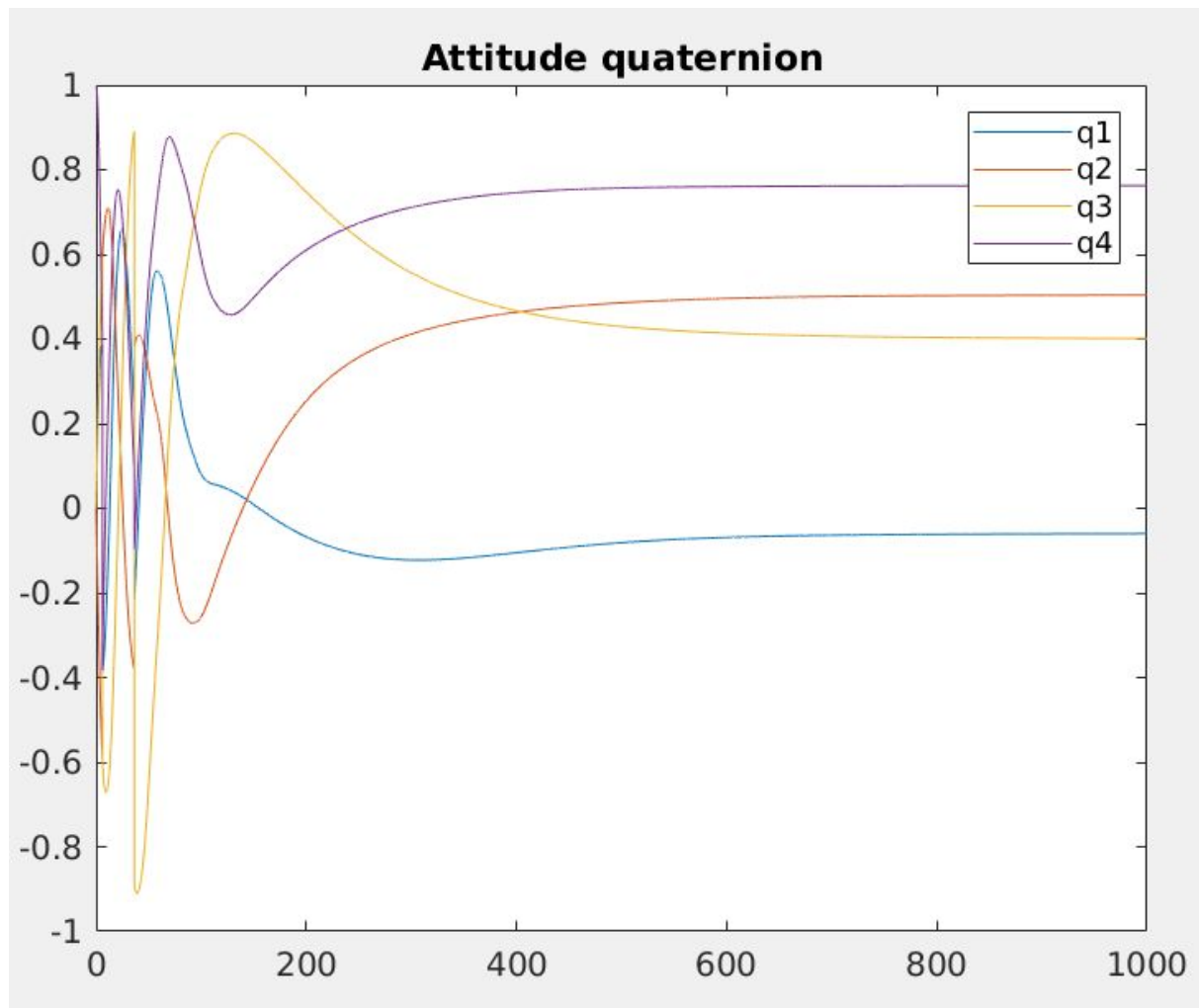
Inner loop

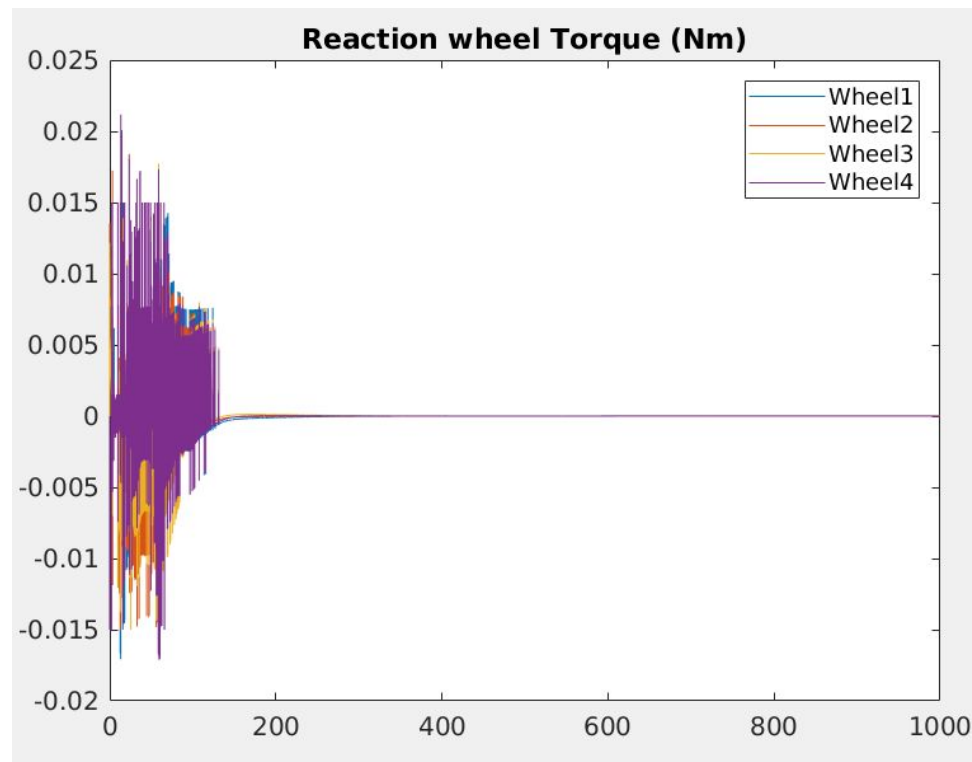
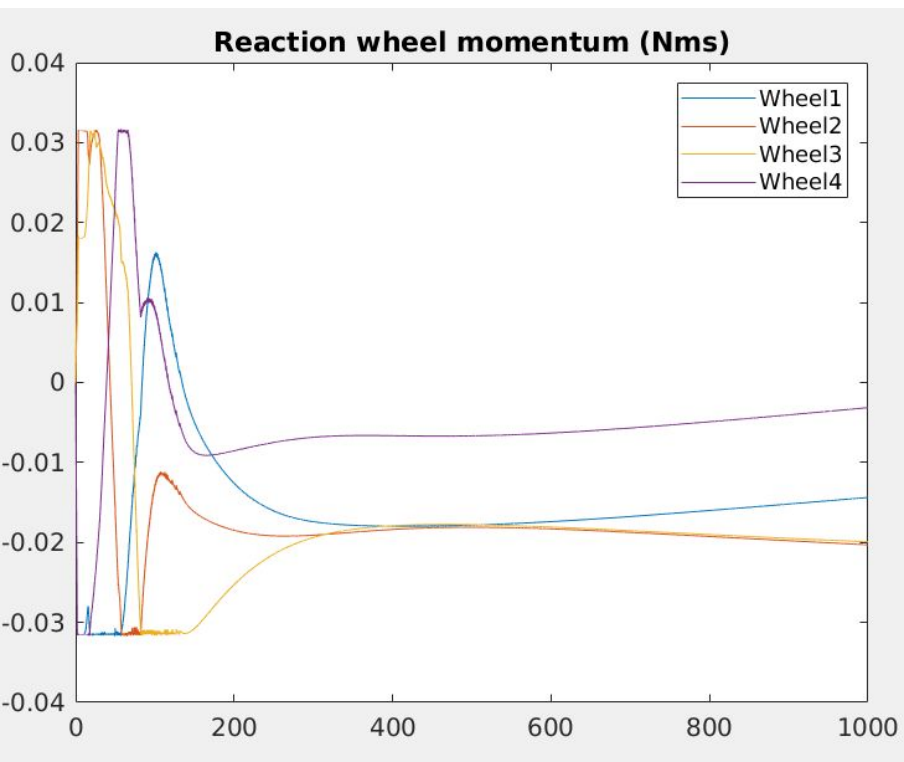
$K_{p_w} = 1.1 \times 10^{-1};$

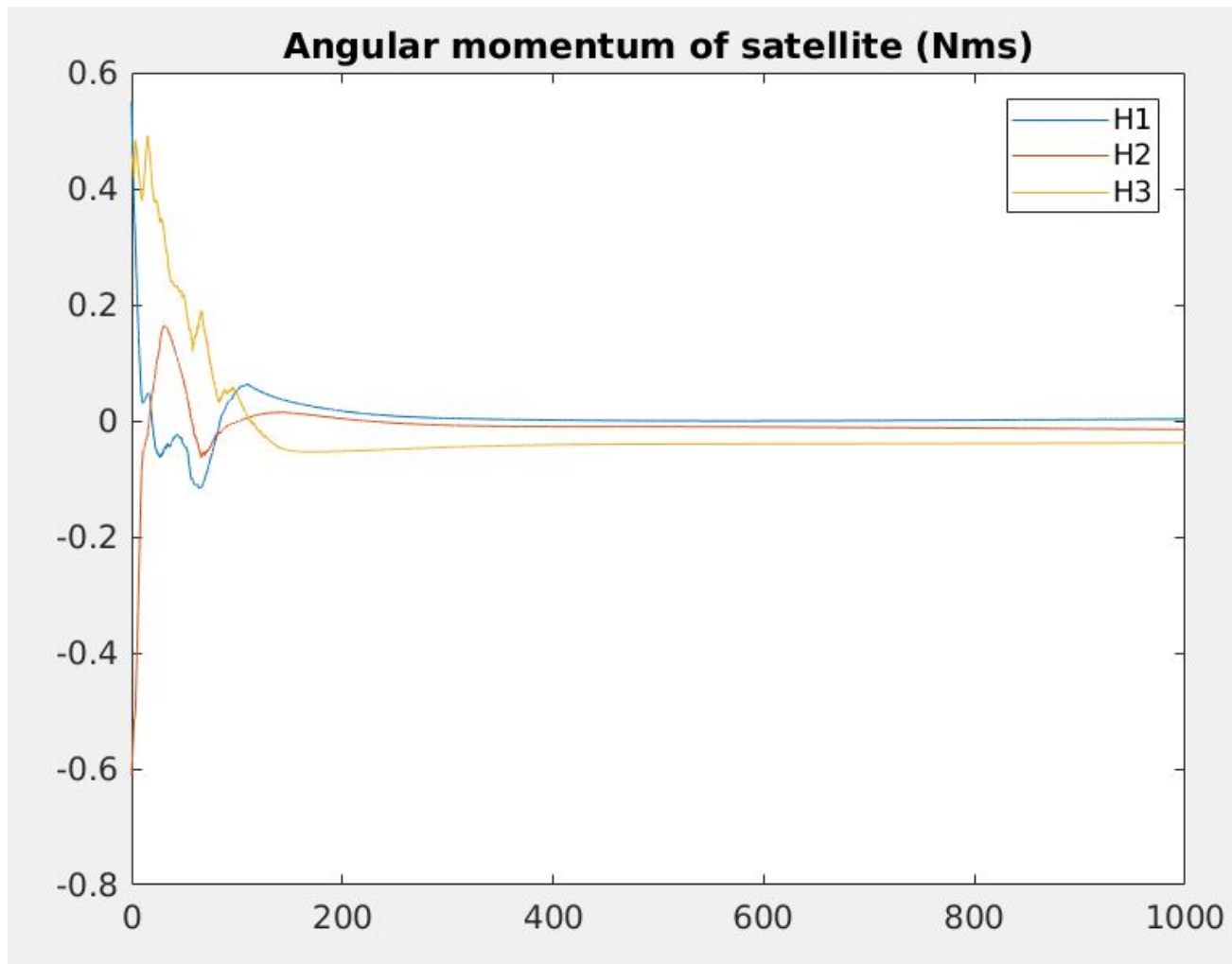
$K_{d_w} = 10^{-1};$

$K_{i_w} = 0$

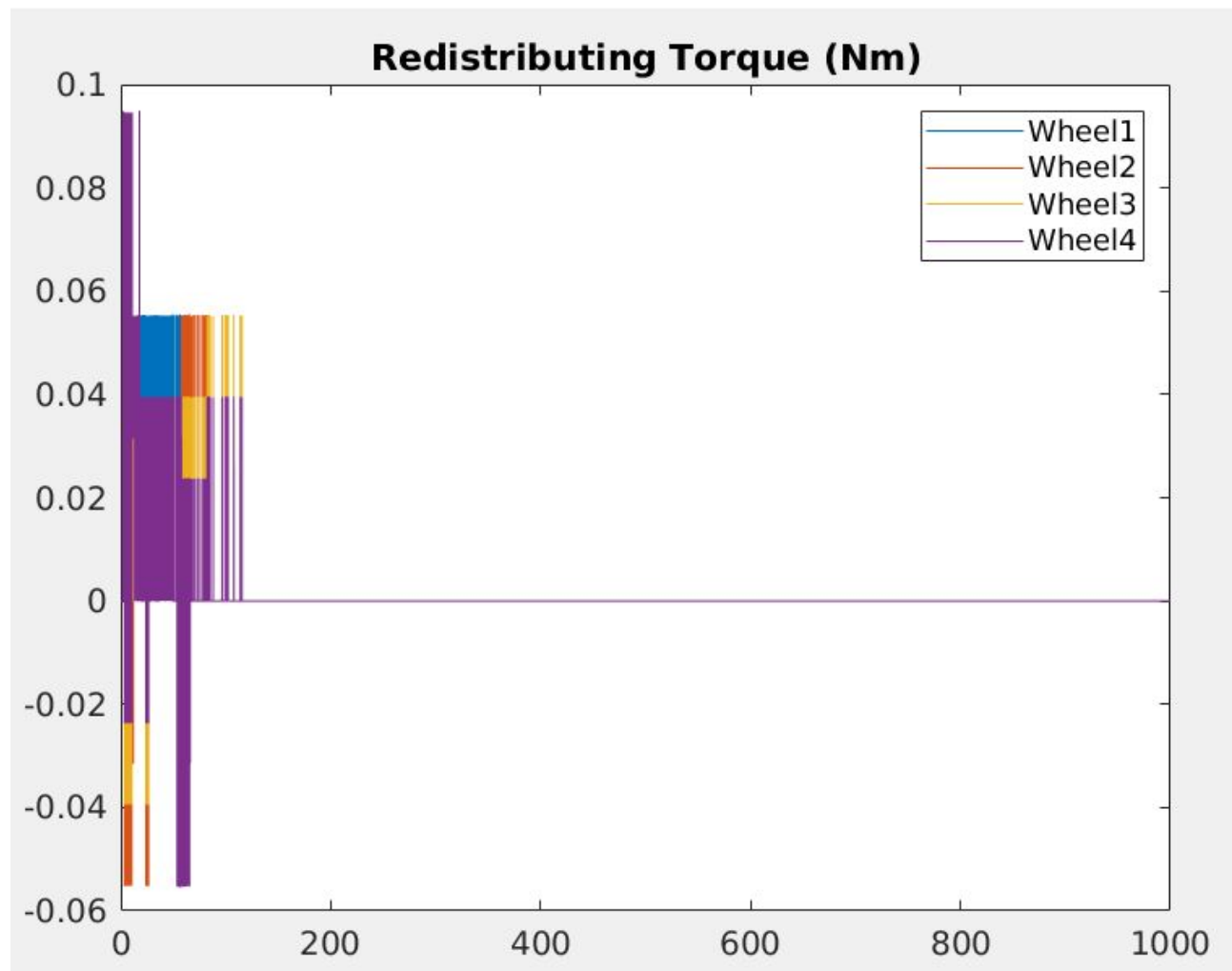




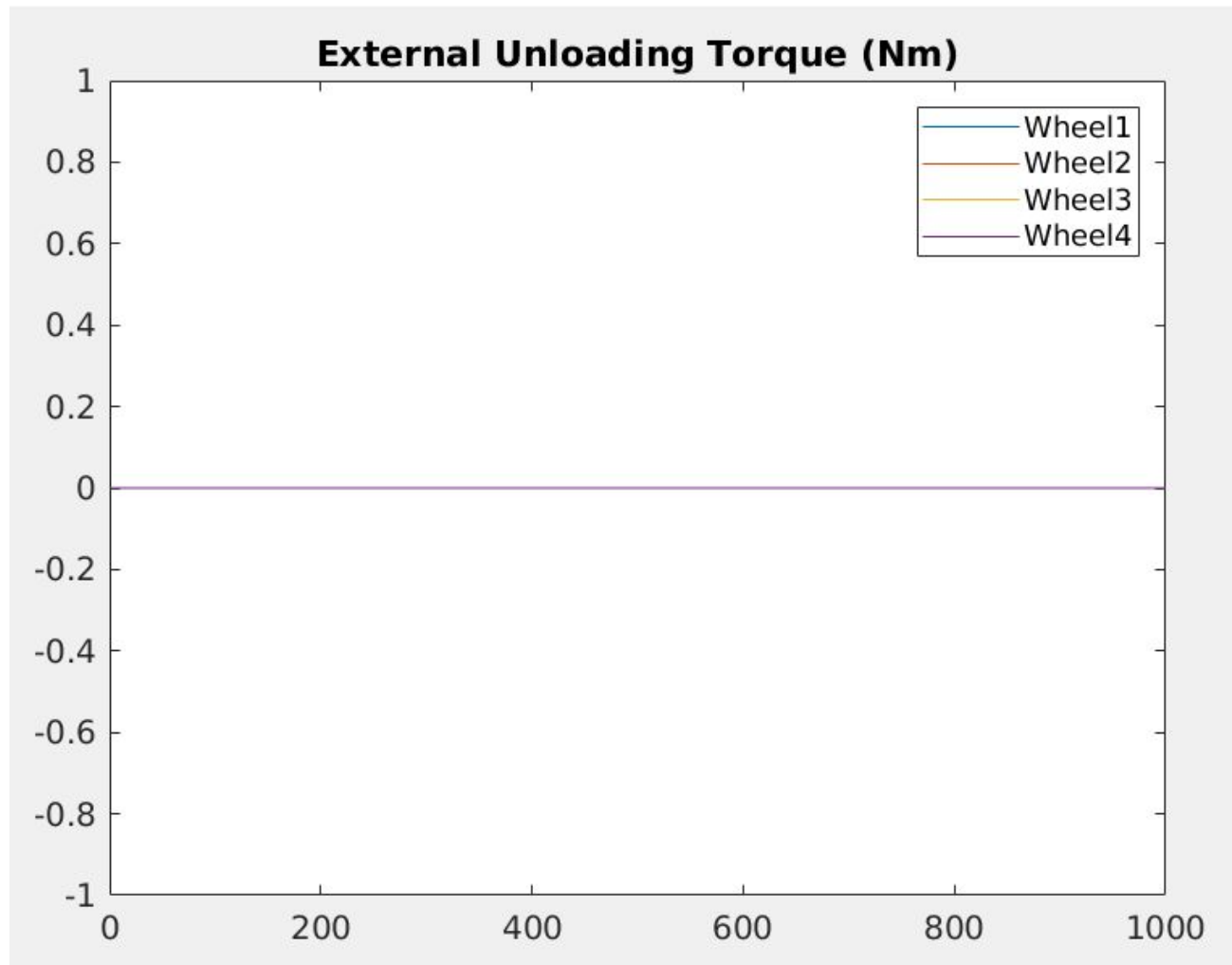




Even though redistribution torque command might seem high, this command passes through the torque saturation block of reaction wheels.



Even though the unloading functionality is implemented, it was not necessary for this problem and tuned control.





## Settling time (2%)

$\text{Theta\_settling\_time} = [667.2000; 500.5000; 504.6000] \text{ s}$

$\text{Omega\_settling\_time} = [595.8000; 766.2000; 817.7000] \text{ s}$

At 900 s, the attitude and angular velocities are

$\text{Theta} = [32.6559; 54.6541; 73.0042] \text{ deg}$

$\text{Omega} = [-0.0317; -0.0411; -0.0321] \text{ deg/s}$





## Ways of improving

- Better momentum management system. The redistribution of momentum does not seem like a smooth operation right now from what is seen on the plot of reaction wheels torques.
- Adding additional blocks that can linearize the dynamics. This will make the system more robust, but also prone to hardware modeling issues. Hence the decision should be made based on the expected error in hardware modeling.
- Add different modes for control operation for different errors in attitude and angular velocity. So different modes for when errors are high, compared to when the errors are low and need finer adjustments.

**Thank You!**