

PAD - Assignment 11. Analysis of Vibrations

$$m \frac{d^2x}{dt^2} + c \frac{dx}{dt} + Rx = F_0 u(t)$$

Taking Laplace transform, we get:

$$m[s^2 \bar{x}(s) - s\bar{x}(0) - \dot{x}(0)] + cs\bar{x}(s) + R\bar{x}(s) = F_0 \bar{u}(s)$$

$$\therefore \bar{x}(s) [ms^2 + cs + R] = F_0 \bar{u}(s)$$

$$\frac{\bar{x}(s)}{\bar{u}(s)} = \frac{F_0}{ms^2 + cs + R}$$

$$\boxed{\frac{\bar{x}(s)}{\bar{u}(s)} = \frac{F_0/k}{\frac{m}{R}s^2 + \frac{c}{R}s + 1}}$$

Comparing with the standard form:

$$\frac{\bar{y}(s)}{\bar{x}(s)} = \frac{K_p}{\tau^2 s^2 + 2\zeta\tau s + 1}$$

we get, $\tau^2 = m/k$

$$\tau = \sqrt{m/k}$$

$$2\zeta\tau = c/k$$

$$\zeta = \frac{c}{2k} \sqrt{\frac{k}{m}} = \frac{c}{2\sqrt{mk}}$$

$$K_p = F_0/k$$

Natural period of oscillation $\tau = \sqrt{m/k}$

Damping coefficient $\xi = \frac{c}{2\sqrt{mk}}$

Steady state gain $= K_p = F_0/k$

Natural Period of Oscillation:

Period of Oscillation

$$T = \frac{1}{2\pi} \left(\frac{2\pi\tau}{\sqrt{1-\xi^2}} \right)$$

for natural oscillation $\xi = 0$

$$T_n = \tau$$

$$T_n = \sqrt{\frac{m}{k}}$$

Damping coefficient:

$$\xi = \frac{c}{2\sqrt{mk}}$$

Steady Gain:

$$K_p = \frac{F_0}{k}$$

(b) Transfer Function:

$$G(s) = \frac{K_p}{\tau s^2 + 2\zeta\tau s + 1}$$

Now, substituting values,

$$G(s) = \frac{F_0/k}{\frac{m}{k}s^2 + \frac{c}{k}s + 1}$$

(c) Underdamped, Overdamped & Critically damped:

We have $G(s) = \frac{F_0/k}{\frac{m}{k}s^2 + \frac{c}{k}s + 1}$

Roots of characteristic equation

$$\frac{m}{k}s^2 + \frac{c}{k}s + 1 = 0$$

$$s = \frac{-c/k \pm \sqrt{(c/k)^2 - 4(m/k)}}{2}$$

For underdamped case, roots are complex conjugate

$$\text{So, } \left(\frac{c}{k}\right)^2 - 4\left(\frac{m}{k}\right) < 0$$

$$c^2 - 4mk < 0$$

$$\frac{c}{2\sqrt{mk}} < 1$$

$$\zeta < 1$$

For critically damped,

$$\left(\frac{c}{k}\right)^2 - 4\left(\frac{m}{k}\right) = 0$$

$$c = 2\sqrt{mk}$$

$$\xi = 1$$

For overdamped case,

$$\left(\frac{c}{k}\right)^2 - \frac{4m}{k} > 0$$

$$c^2 > 4mk$$

$$\frac{c}{2\sqrt{mk}} > 1$$

$$\xi > 1$$

(d) Response for step input

$$u(s) = A/s$$

For underdamped

$$\bar{y}(s) = \frac{1}{s} \left[\frac{F_0/k}{\tau^2 s^2 + 2\xi\tau s + 1} \right]$$

$$\bar{y}(s) = \frac{AF_0}{k} \cdot \frac{1}{\tau^2} \left[\frac{1}{s} - \frac{s + 2\xi/\tau}{s^2 + 2\xi\tau s + \frac{1}{\tau^2}} \right] \tau^2$$

Taking inverse Laplace

$$Y(t) = \frac{AF_0}{R} \left[1 - e^{-\xi t/\tau} \left[\sqrt{1-\xi^2} \cos \sqrt{1-\xi^2} \frac{t}{\tau} + \xi \sin \sqrt{1-\xi^2} \frac{t}{\tau} \right] \right]$$

$$p \cos A + q \sin A = r \sin(A + \phi)$$

$$r = \sqrt{p^2 + q^2} \quad \phi = \tan^{-1} \left(\frac{p}{q} \right)$$

$$Y(t) = \frac{AF_0}{R} \left[1 - \frac{e^{-\xi t/\tau}}{\sqrt{1-\xi^2}} \sin(\omega t + \phi) \right]$$

$$\omega = \frac{\sqrt{1-\xi^2}}{\tau}$$

Critically damped ($\xi = 1$)

$$y(s) = \frac{A}{s} \cdot \frac{F_0/R}{\tau^2 s^2 + 2\xi\tau s + 1}$$

$$y(s) = \frac{AF_0}{R} \cdot \frac{1}{\tau^2} \left[\frac{1}{s(s + 1/\tau)^2} \right]$$

By partial fraction,

$$y(s) = \frac{AF_0}{R} \left[\frac{1}{s} - \frac{1}{s + 1/\tau} - \frac{1/\tau}{(s + 1/\tau)^2} \right]$$

By Inverse Laplace

$$Y(t) = \frac{AF_0}{R} \left[1 - e^{-t/\tau} \left(1 + \frac{t}{\tau} \right) \right]$$

Overdamped Response ($\xi > 1$)

$$\bar{y}(s) = \frac{A(F_0)}{s} \left(\frac{1}{R} \right) \left(\frac{1}{\tau^2} \right) \cdot \frac{1}{\left(s^2 + 2\frac{\xi}{\tau} s + \frac{1}{\tau^2} \right)}$$

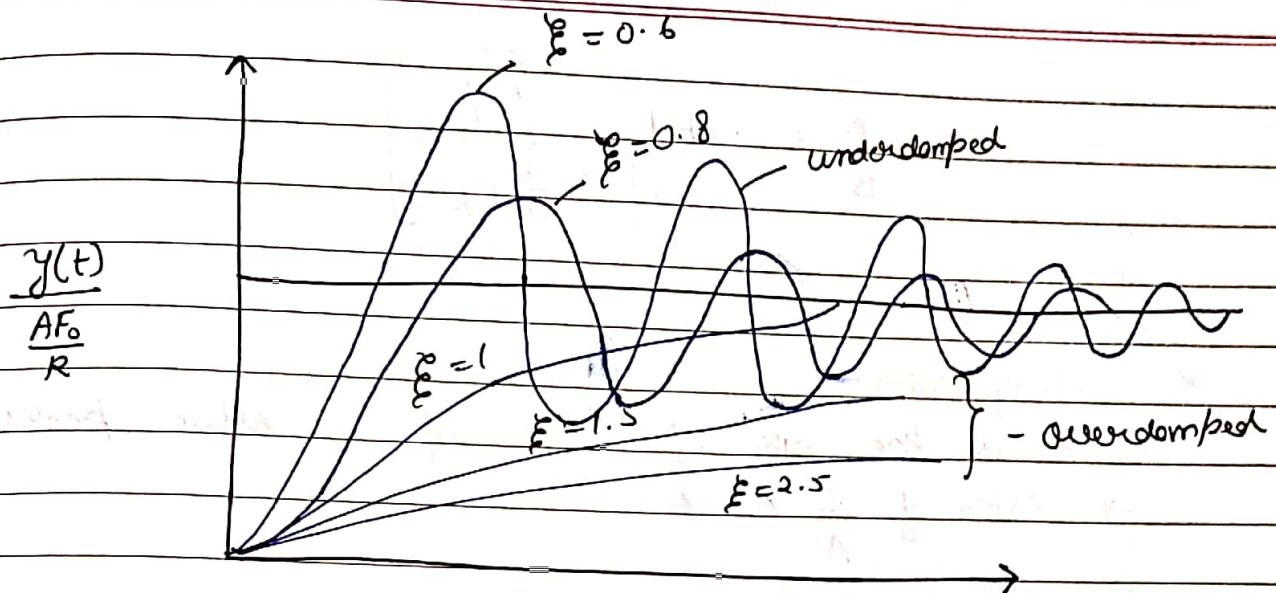
$$\bar{y}(s) = \left(\frac{AF_0}{R} \right) \left[\frac{1}{s} - \frac{s + \xi/\tau}{\left(s + \frac{\xi}{\tau} \right)^2 - \left(\frac{\xi^2 - 1}{\tau^2} \right)} - \frac{\xi/\tau \sqrt{\xi^2 - 1} \sqrt{\frac{\tau^2}{\xi^2 - 1}}}{\left(s + \xi/\tau \right)^2 - \left(\frac{\xi^2 - 1}{\tau^2} \right)^2} \right]$$

Taking Inverse Laplace

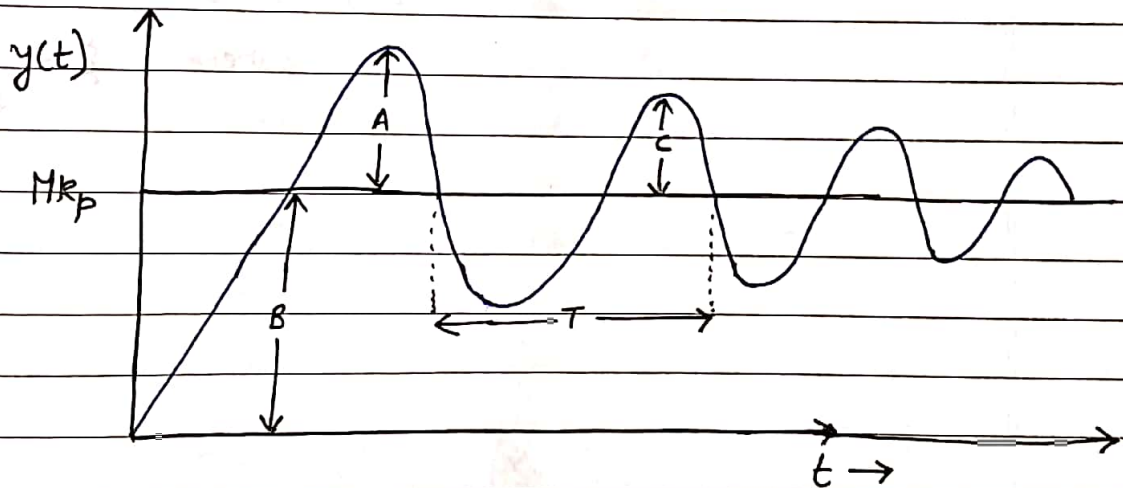
$$y(t) = \frac{AF_0}{R} \left[1 - e^{-\xi t/\tau} \cosh \sqrt{\frac{\xi^2 - 1}{\tau^2}} t + \frac{\xi}{\sqrt{\xi^2 - 1}} \sinh \sqrt{\frac{\xi^2 - 1}{\tau^2}} t \right]$$

- * For underdamped system, decreasing ξ makes the response more oscillatory
- * For Critically system, response reaches steady state more quickly without oscillations
- * For overdamped system, as ξ increases, the response becomes more sluggish.

$\frac{1}{\xi} \propto \frac{1}{\text{speed of response}}$
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(c) Under-damped Response



(i) Overshoot:

→ measure of how much response exceeds the ultimate value following a step change

→ Ratio of A/B

$$\text{Overshoot} = \frac{A}{B} = \exp \left[\frac{-\pi \zeta}{\sqrt{1-\zeta^2}} \right]$$

$$\zeta = \frac{c}{2\sqrt{mK}}$$

$$\frac{A}{B} = \exp \left[\frac{-\frac{\pi c}{2\sqrt{mk}}}{\sqrt{1 - \frac{c^2}{4mk}}} \right]$$

2. Decay Ratio:

It is the ratio of sizes of successive peaks

→ Ratio of $\frac{C}{A}$

$$\text{Decay Ratio} = \frac{C}{A} = \exp \left[\frac{-2\pi \xi}{\sqrt{1 - \xi^2}} \right] = (\text{overshoot})^2$$

where $\xi = \frac{c}{2\sqrt{mk}}$

Response of a body towards changes in reservoir temperature
 Given Equation

$$\frac{dT}{dt} = -h(T - T_{\infty})$$

$T_{\infty} \rightarrow$ reservoir temp

$T_0 \rightarrow$ at $t=0$

$$\frac{dT}{dt} = -hT + hT_{\infty}$$

$$\frac{dT}{dt} + hT = hT_{\infty}$$

$$\frac{1}{h} \cdot \frac{dT}{dt} + T = T_{\infty} \rightarrow (i)$$

For, 1st order system

$$\tau \cdot \frac{dy}{dt} + y = X(t) \cdot K_p \rightarrow (ii)$$

Comparing (i) & (ii)

$$\tau = 1/h$$

$$K_p = 1$$

Time constant Steady state gain

$$G(s) = \frac{1}{(\frac{1}{h})s + 1}$$

(a) Step input of magnitude A

$$G(s) = \frac{T'(s)}{T_{\infty}'(s)} = \frac{1}{\frac{1}{h}s + 1}$$

$$T'(s) = T(t) - T_s$$

$$T_{\infty}'(s) = T_{\infty}(t) - T_s$$

$$T_{\infty}'(s) = A/s$$

$$\tau = 1/h$$

$$T'(s) = \frac{A}{s} \left(\frac{1}{\frac{1}{h}s + 1} \right) \quad k=1$$

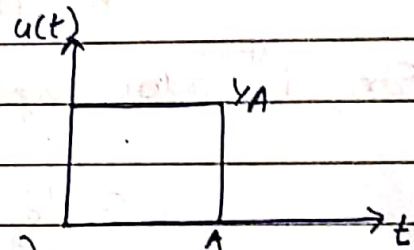
$$T(t) = T_{\infty} + A(1 - e^{-ht})$$

(b) Pulse function:

$$\delta = \begin{cases} 1/A & 0 \leq t \leq A \\ 0 & \text{otherwise} \end{cases}$$

$$T_{\infty}'(s) = \frac{1}{A} \frac{(1 - e^{-As})}{s}$$

$$T'(s) = \frac{(1 - e^{-As})}{As} \left(\frac{1}{\frac{1}{h}s + 1} \right)$$



By partial fraction

$$T'(s) = \frac{1}{A} \left[\left(\frac{1}{s} - \frac{1}{s+h} \right) - \left(\frac{1}{s} - \frac{1}{s+h} \right) e^{-As} \right]$$

Inverse Laplace

$$T = T_{\infty} + \frac{1}{A} \left[(1 - e^{-ht}) - (1 - e^{-h(t-A)}) u(t-A) \right]$$

Impulse function:

Impulse function = $\delta(t)$

$$T_{\infty}'(s) = 1$$

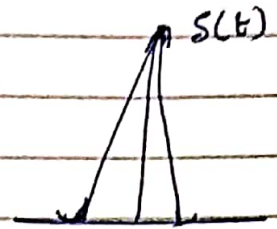
$$T'(s) = \frac{1}{\frac{1}{h}s + 1}$$

Taking inverse laplace

$$T'(t) = h e^{-ht}$$

$$\tau = \frac{1}{h}$$

$$T(t) = T_d + h e^{-ht}$$

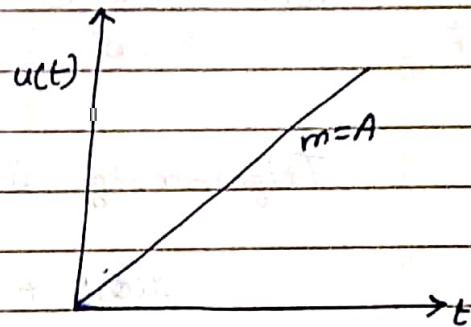


Ramp Function:

$$u(t) = At$$

$$T'(s) = A/s^2$$

$$T'(s) = \frac{A}{s^2} \left(\frac{1}{\frac{1}{\tau}s + 1} \right)$$



$$\tau = 1/h$$

$$T'(s) = \frac{A}{\tau} \left[\frac{\tau^2}{s + 1/\tau} + \frac{-\tau^2}{s} + \frac{\tau}{s^2} \right]$$

By laplace inverse

$$T(t) = -T_d = A(t - \tau(1 - e^{-t/\tau})) \quad \text{where } \tau = \frac{1}{h}$$

For sinusoidal response

$$T_{\infty}'(t) = A \sin \omega t$$

$$T_{\infty}'(s) = \frac{A\omega}{s^2 + \omega^2}$$

$$T'(s) = \frac{A\omega}{s^2 + \omega^2} \left(\frac{1/\tau}{s + 1/\tau} \right)$$

By partial fraction

$$T'(s) = \frac{A\omega}{\tau} \left[\frac{\tau^2}{(1+\tau^2\omega^2)(s+\frac{1}{\tau})} - \frac{c^2 s}{(1+\tau^2\omega^2)(s^2+\omega^2)} + \frac{\tau}{(1+\tau^2\omega^2)(s^2+\omega^2)} \right]$$

Taking Laplace inverse

$$T(t) = \frac{A\omega\tau}{1+\tau^2\omega^2} e^{-t/\tau} - \frac{A\omega\tau\cos\omega t}{1+\tau^2\omega^2} + \frac{A\sin\omega t}{1+\tau^2\omega^2}$$

Trigonometry Identity

$$a\cos\theta + b\sin\theta = r\sin(\theta + \phi)$$

For this case, $r = \frac{A}{\sqrt{1+\tau^2\omega^2}}$ $\phi = \tan^{-1}(\omega\tau)$

$$T(t) = T_{\infty} + \frac{A\omega\tau e^{-t/\tau}}{1+\tau^2\omega^2} + \frac{A}{\sqrt{1+\tau^2\omega^2}} \sin(\omega t + \phi)$$

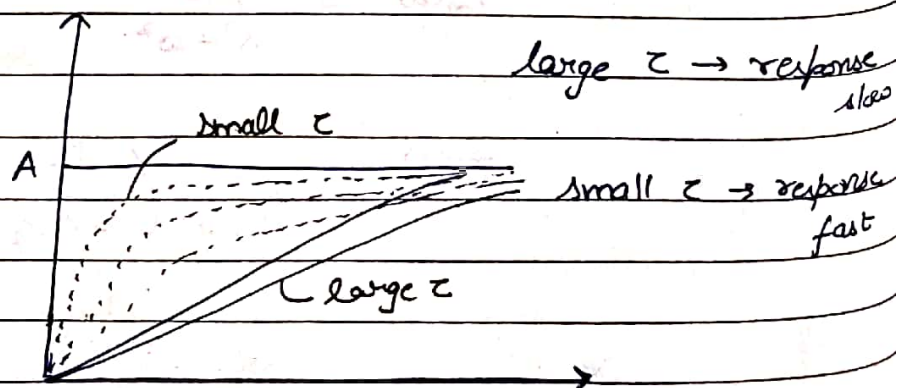
$$\tau = 1/h$$

Effect of Time constant

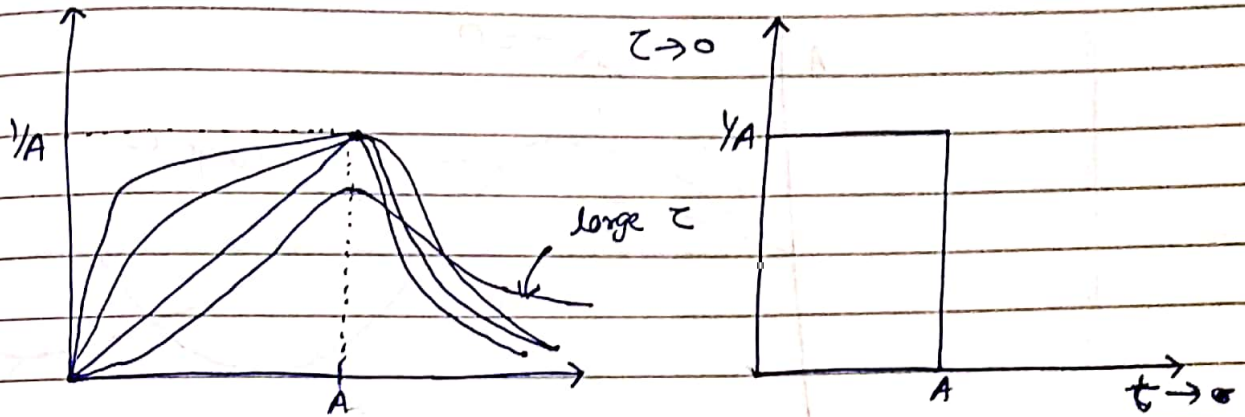
C \rightarrow very small time constant

D \rightarrow extremely large time constant

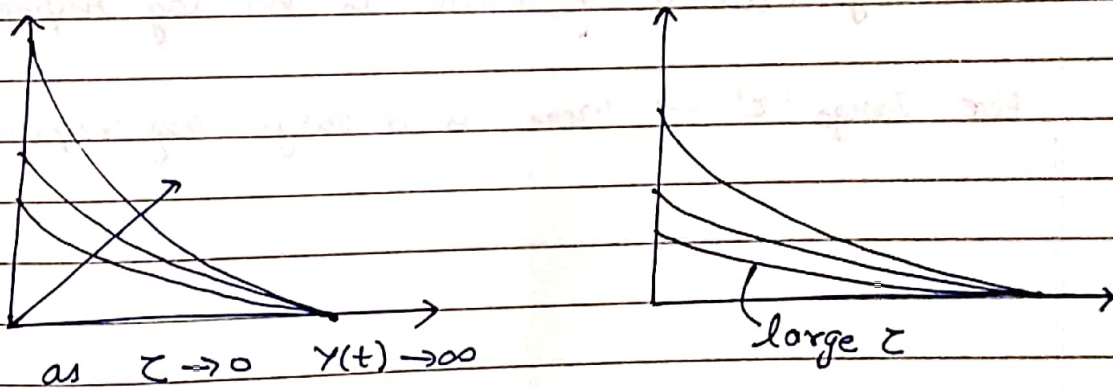
(i) For step input



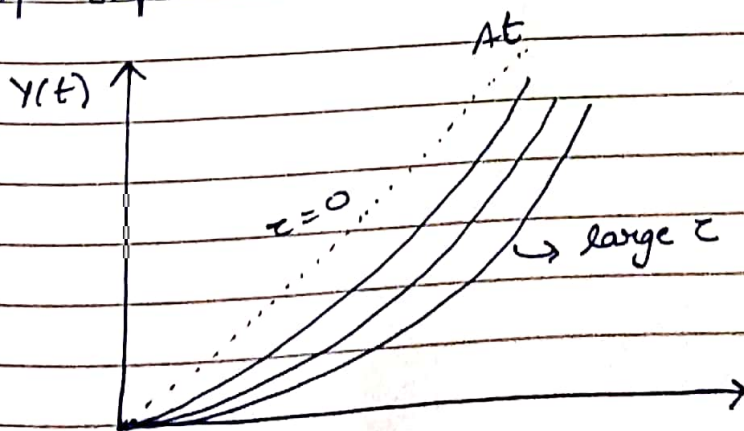
(ii) For Pulse



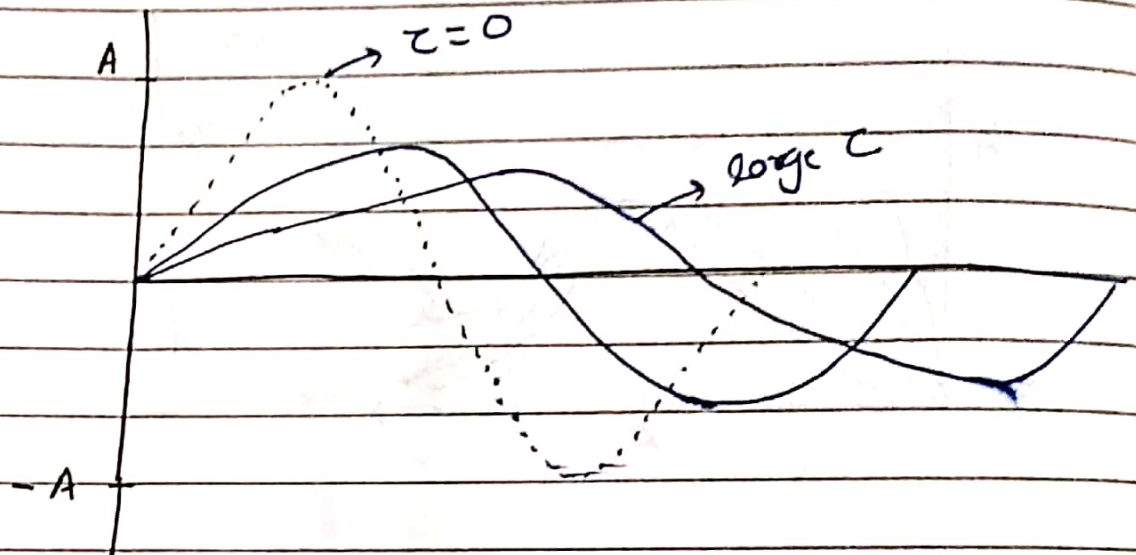
(iii) Unit Impulse



(iv) Ramp Input



(v) Sinusoidal Input



For very small ' τ ' \rightarrow There is no lag response.

For large ' τ ' \rightarrow There is a large lag response.