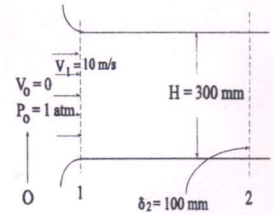


Flow of air ( $\rho = 1.23 \text{ kg/m}^3$ ) develops in a flat horizontal duct following a well rounded entrance section. The duct height is  $H = 300 \text{ mm}$ . Turbulent boundary layer grows on the duct walls, but the flow is not yet fully developed. Assume that the velocity profile in each boundary layer is given by  $u/U = (y/\delta)^{1/7}$ . The inlet flow is uniform with  $V = 10 \text{ m/s}$  at section 1. At section 2, the boundary layer thickness on each wall of the channel is  $\delta_2 = 100 \text{ mm}$ . Show that for this flow,  $\delta^* = \delta/8$ . Evaluate the static gauge pressure at section 2. Find the average wall shear stress between the entrance and section 2 at  $L = 5 \text{ m}$ . The region outside the entrance (location O in figure) can be taken to be still air with pressure equal to atmospheric pressure.



(3)

$$\delta^* = \int_0^\delta \left(1 - \frac{u}{U}\right) dy = \delta \int_0^1 \left(1 - \frac{u}{U}\right) d\eta = \delta \int_0^1 (1 - \eta^{1/7}) d\eta$$

$$= \delta \left( \eta - \frac{7}{8} \eta^{8/7} \right) \Big|_0^1$$

$$\boxed{\delta^* = \delta/8}$$

From continuity  $V_1 A_1 = V_2 A_2 = V_2 w (H - 2\delta_2^*)$

$$V_2 = V_1 \frac{H}{H - 2\delta_2^*} = 10 \frac{\text{m}}{\text{s}} \frac{300 \text{ mm}}{(300 - 25) \text{ mm}} = 10.9 \text{ m/s}$$

From Bernoulli, since  $z = \text{const}$ .

$$\frac{p_0}{\rho} + \frac{V_0^2}{2} = \frac{p}{\rho} + \frac{V^2}{2}$$

$$p_{1g} = p_1 - p_0 = -\frac{1}{2} \rho V_1^2 = -\frac{1}{2} \times 1.23 \frac{\text{kg}}{\text{m}^3} \times 10^2 \frac{\text{m}^2}{\text{s}^2} \times \frac{\text{N s}^2}{\text{kg m}}$$

$$p_{1g} = -61.5 \text{ Pa}$$

$$p_{2g} = p_2 - p_0 = -\frac{1}{2} \rho V_2^2 = -\frac{1}{2} \times 1.23 \times (10.9)^2 = -73.1 \text{ Pa}$$

Apply momentum using cv. (half the duct)

$$F_{s2} + F_{B2} = \frac{\partial}{\partial t} \int_{cv} u \rho dV + \int_{cs} u \rho \vec{V} \cdot d\vec{A}$$

$$(p_1 - p_2) w \frac{H}{2} - \tau_w L = \bar{V}_1 \left\{ -p \bar{V}_1 \cdot \frac{H}{2} w \right\} + \int_0^H u \rho w dy$$

$$+ V_2 \left\{ + p V_2 \left( \frac{H}{2} - \delta_2 \right) w \right\}$$

$$A = \rho V_2^2 \delta_2 w \int_0^1 \eta^{2/7} d\eta = \rho V_2^2 \frac{7}{9} \delta_2 w$$

$$\therefore \bar{T}_{WL} = (p_1 - p_2) w \frac{H}{2} + \rho v_1^2 \frac{H}{2} w - \rho v_2^2 \left( \frac{H}{2} - \frac{2}{9} s_2 \right) w$$

$$\bar{T} = \frac{1}{L} \left( (p_1 - p_2) \frac{H}{2} + \rho v_1^2 \frac{H}{2} - \rho v_2^2 \left( \frac{H}{2} - \frac{2}{9} s_2 \right) \right)$$

$$\bar{T} = \frac{1}{5} \left[ 11.6 \times 0.15 + 1.23 \times 10^2 \times 0.15 - 1.23 \times (10.9)^2 (0.15 - 0.022) \right]$$

$$\bar{T} = 0.3 \text{ N/m}^2$$

$$\frac{(p_1 - p_2) H}{2} =$$

$$\frac{2}{2} = 1$$

$$(25 - 4) w_{clv} \cdot A_{clv} = H w_{clv} \cdot A_{clv} \text{ pressure exerted}$$

$$2/2 \times 12 = \frac{1000 \times 1000}{4 \times (25 - 20) \times 10^3} \times \frac{11}{25} \times 25 \times 4 \times 10^3$$

$$25 - 20 = 5 \text{ cm} = 0.05 \text{ m}$$

$$\frac{p_1}{\rho} + \frac{v_1^2}{2} = \frac{p_2}{\rho} + \frac{v_2^2}{2}$$

$$\frac{2000 \times 10^3}{1000} + \frac{10^2}{2} = \frac{p_2}{1000} + \frac{1}{2} \times \frac{1}{5} = 200 - p_2 = 199.5$$

$$p_2 = 0.5 \text{ Pa}$$

$$0.9/0.05 = (p_{atm}) \times 0.05 \times L = \frac{5}{2} \times 10^3 \times \frac{1}{5} = 1000 \text{ Pa}$$

$$\frac{1}{A} \frac{dA}{dt} = \frac{1}{L} \frac{dL}{dt} = \frac{1}{V} \frac{dV}{dt} = \frac{1}{\rho} \frac{d\rho}{dt} = \frac{1}{\rho} \frac{dp}{dt}$$

$$\frac{1}{A} \frac{dA}{dt} = \frac{1}{L} \frac{dL}{dt} = \frac{1}{V} \frac{dV}{dt} = \frac{1}{\rho} \frac{d\rho}{dt} = \frac{1}{\rho} \frac{dp}{dt}$$

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