



PennState
College of Engineering

ME 563: Nonlinear Finite Elements

Application of the Finite Element
Method to Real World Problems
Application: Ball-Plate Impact with “Failure”

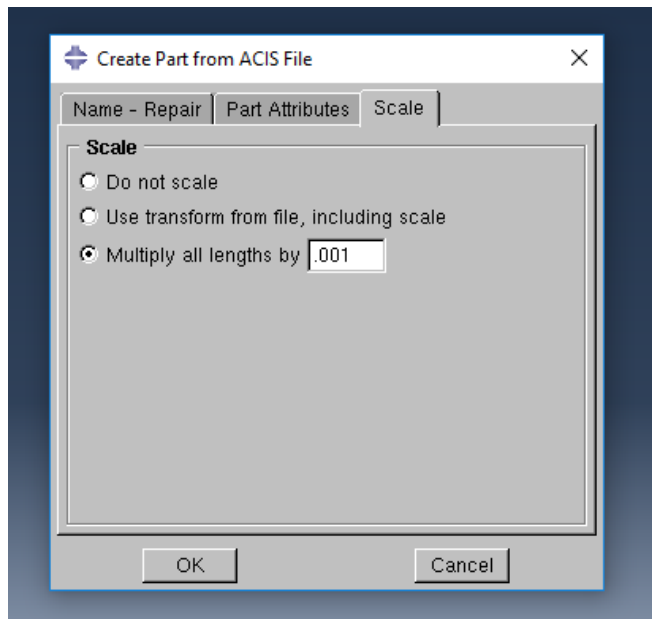
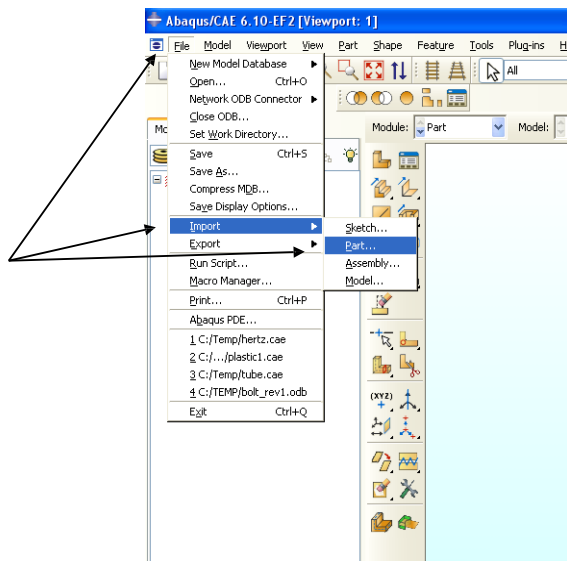
This tutorial accompanies covers a basic example of a ball being fired at an aluminum plate. An element deletion criterion is defined and therefore the plate ruptures and allows the ball to pass through

1. Geometry Import

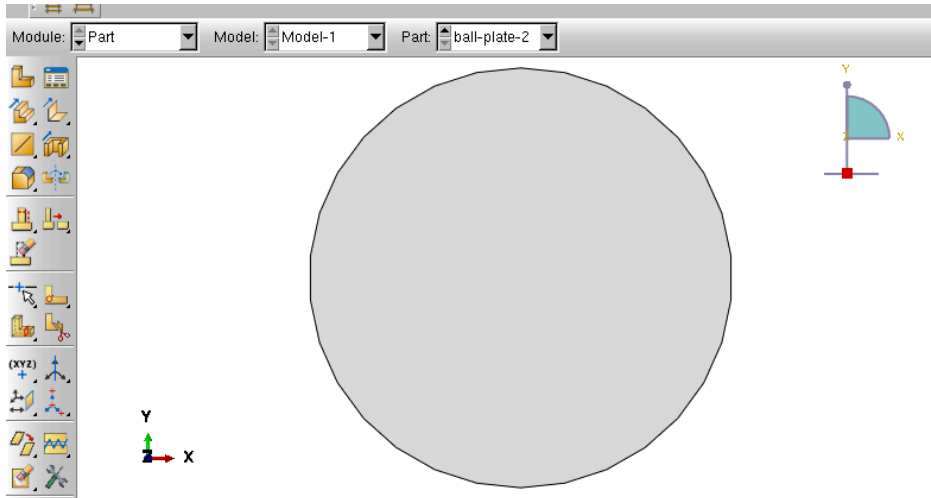
Import the geometry in the form of a step file – **ball-plate.sat** located on Github.

```
git clone https://github.com/rhk12/ball-impact
```

File-> Import-> Part, Scale the part by 0.001 to make it in meters.

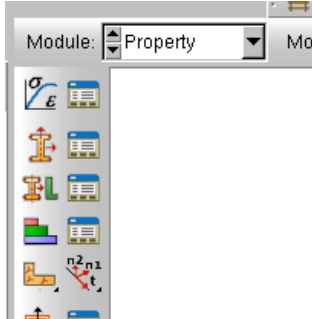


Should look like this:



2. Material and section properties

Start to define the material properties – go into the properties module



Define two materials – **Steel** which will only have linear properties and **Aluminum** which will have yield and post yield behavior.

As the problem is dynamic we also need to define a density for both.

1. **Steel** – Young’s Modulus = 210×10^9 Pa and Poisson’s Ratio = 0.3,
density = 7800 kg/m^3

Edit Material [X]

Name: steel

Description: []

Material Behaviors

Density

Elastic

General Mechanical Thermal Electrical/Magnetic Other

Elastic

Type: Isotropic [Suboptions]

☐ Use temperature-dependent data

Number of field variables: 0

Moduli time scale (for viscoelasticity): Long-term

☐ No compression

☐ No tension

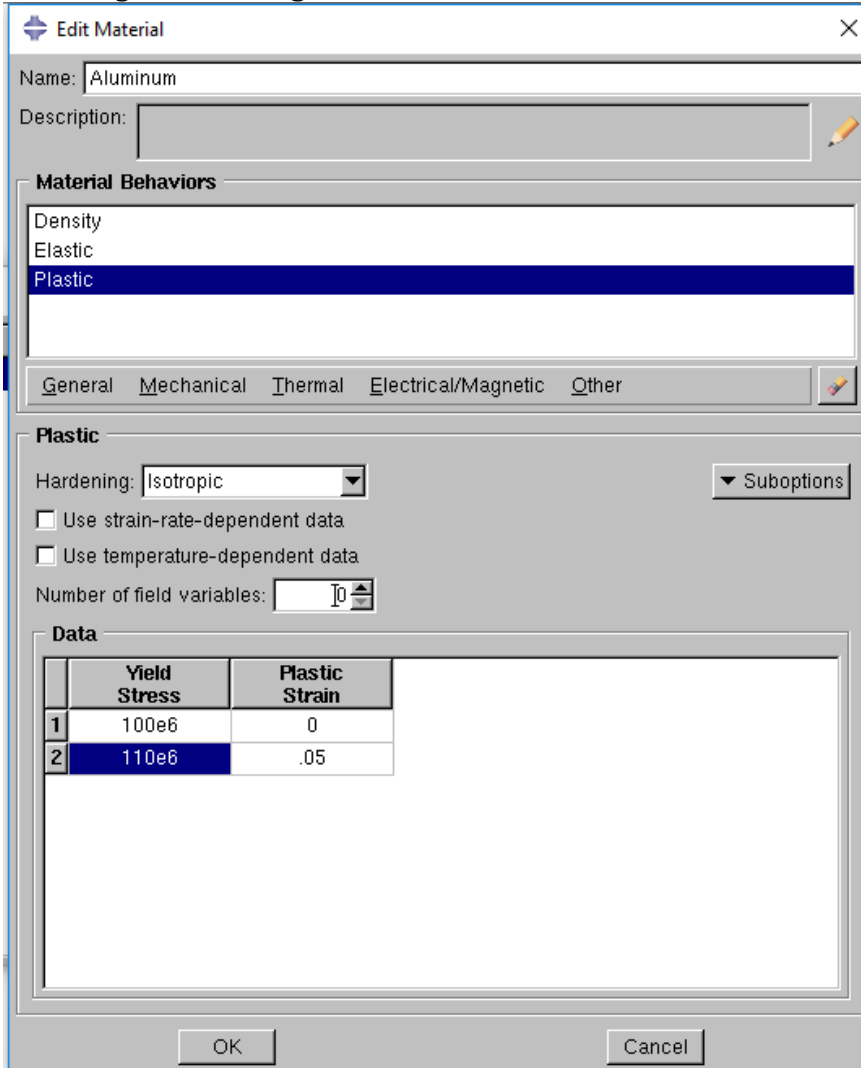
Data

	Young's Modulus	Poisson's Ratio
1	210e9	.3

OK Cancel

2. **Aluminum** – Young's Modulus = 70×10^9 Pa and Poisson's Ratio = 0.3,
density = 2500 kg/m^3

A simple bi-linear material plasticity definition is used. Initial yield stress is set at 100×10^6 Pa with some slight hardening.



Edit Material

Name: Aluminum

Description:

Material Behaviors

Density
Elastic
Plastic

General Mechanical Thermal Electrical/Magnetic Other

Plastic

Hardening: Isotropic Suboptions

☐ Use strain-rate-dependent data
☐ Use temperature-dependent data

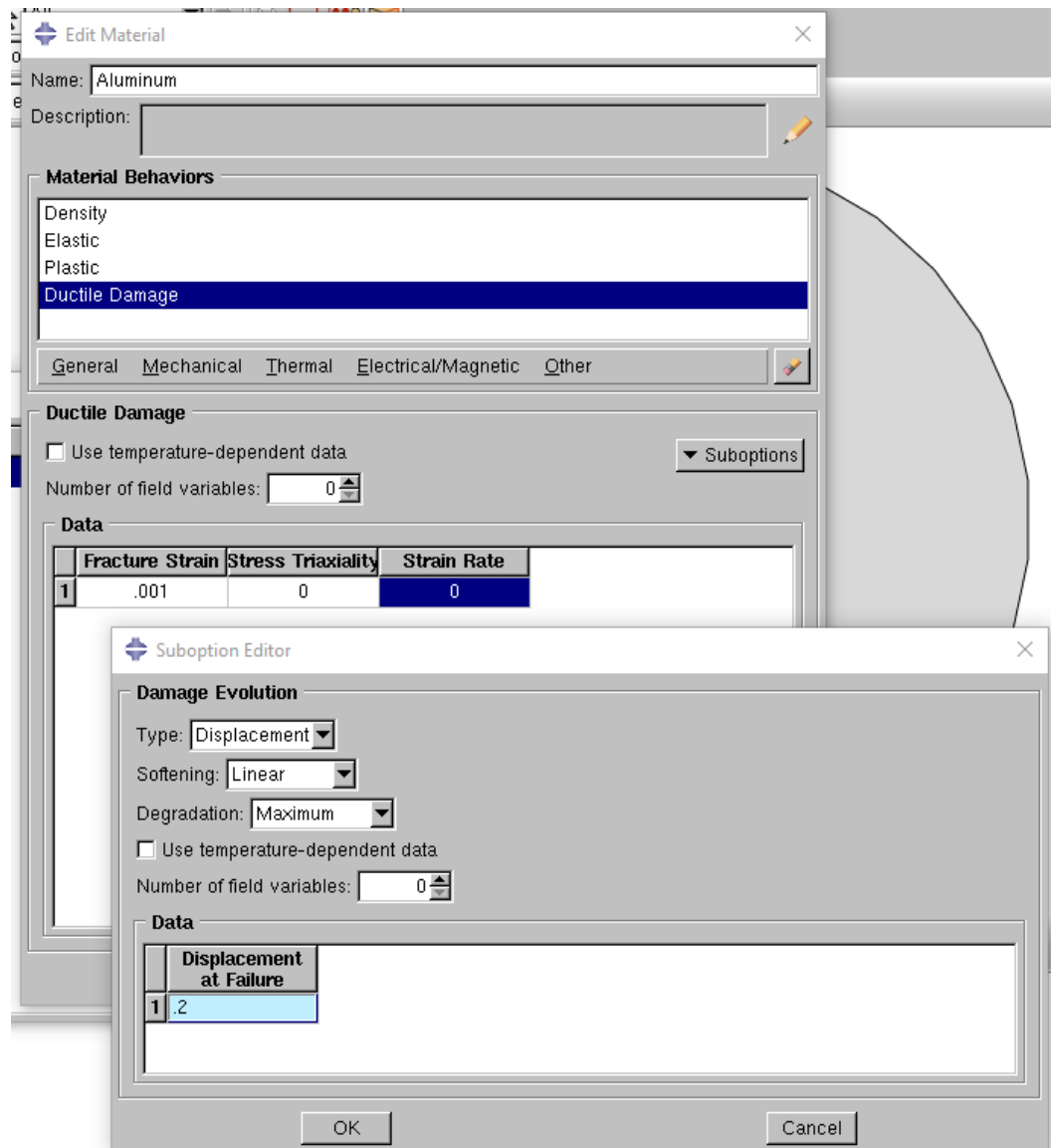
Number of field variables: 10

Data

	Yield Stress	Plastic Strain
1	100e6	0
2	110e6	.05

OK Cancel

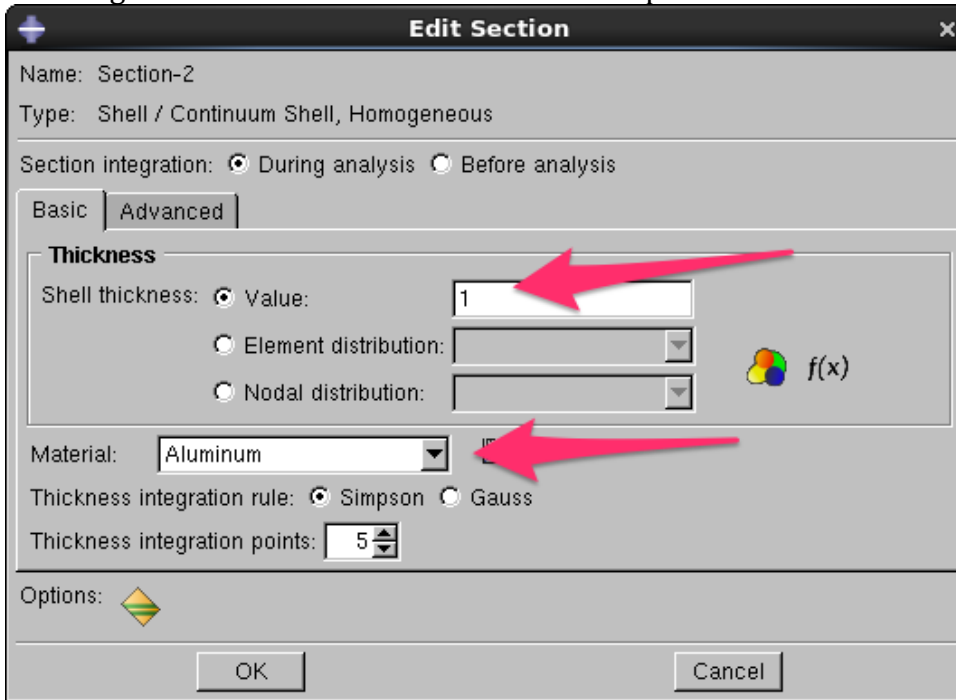
Add the Ductile Damage option. Element deletion criteria: Displacement at failure (damage evolution) is defined using the Suboption button.




3. Define Sections

A simple steel section (solid., homogeneous) is defined and applied to the ball.

A homogeneous shell section is defined for the plate.

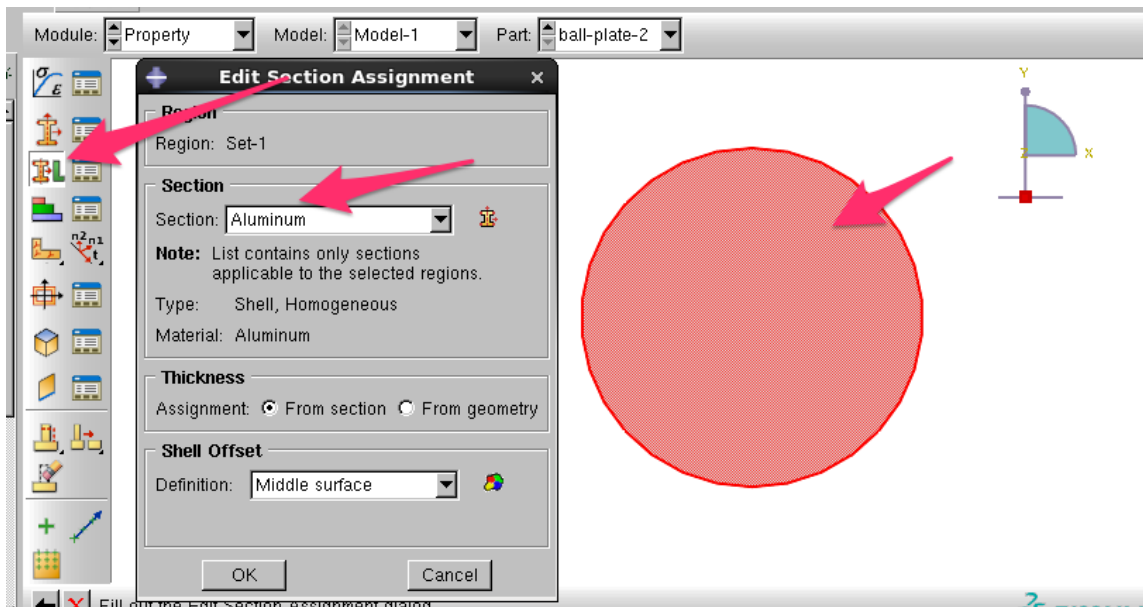


The image shows a software dialog box titled "Edit Section". It contains the following fields and options:

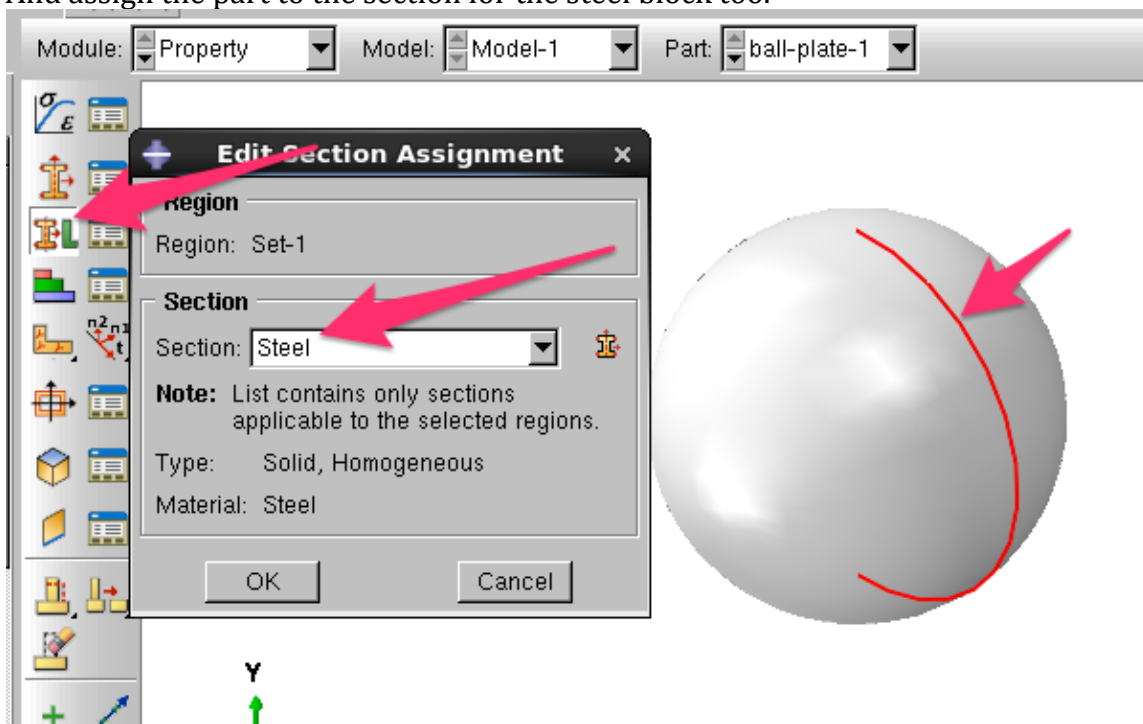
- Name: Section-2
- Type: Shell / Continuum Shell, Homogeneous
- Section integration: ☒ During analysis ☐ Before analysis
- Basic | Advanced (tabbed interface, Basic is selected)
- Thickness** section:
 - Shell thickness: ☒ Value: (indicated by a red arrow)
 - ☐ Element distribution: (with a color gradient icon)
 - ☐ Nodal distribution: (with a color gradient icon)
- Material: (indicated by a red arrow)
- Thickness integration rule: ☒ Simpson ☐ Gauss
- Thickness integration points:
- Options: 
- Buttons: OK, Cancel

Assign sections:

For Al:

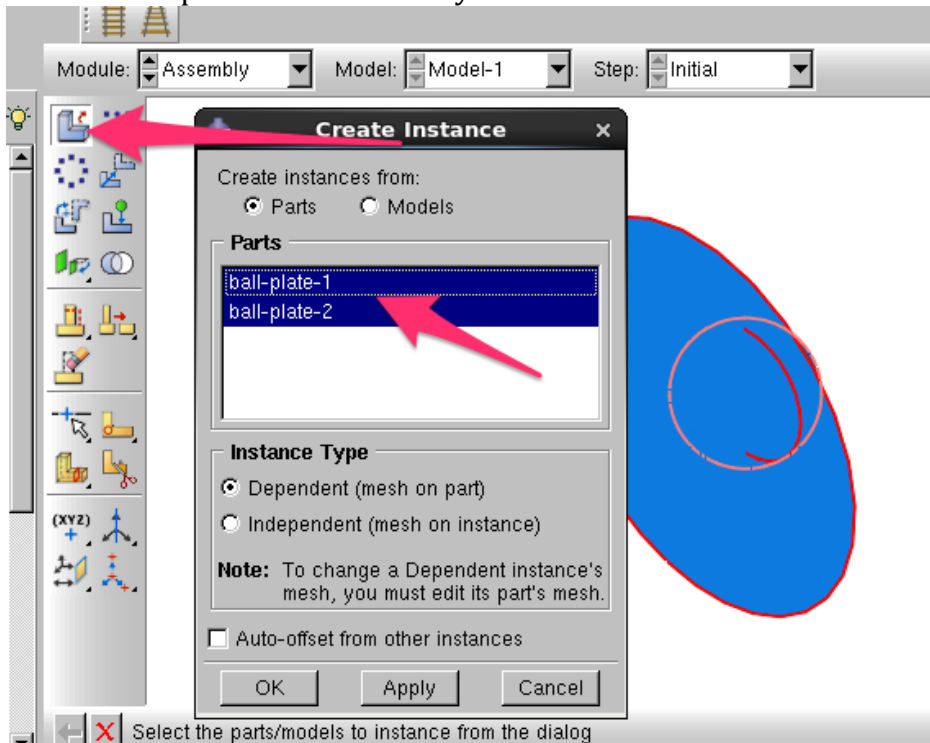


And assign the part to the section for the steel block too.



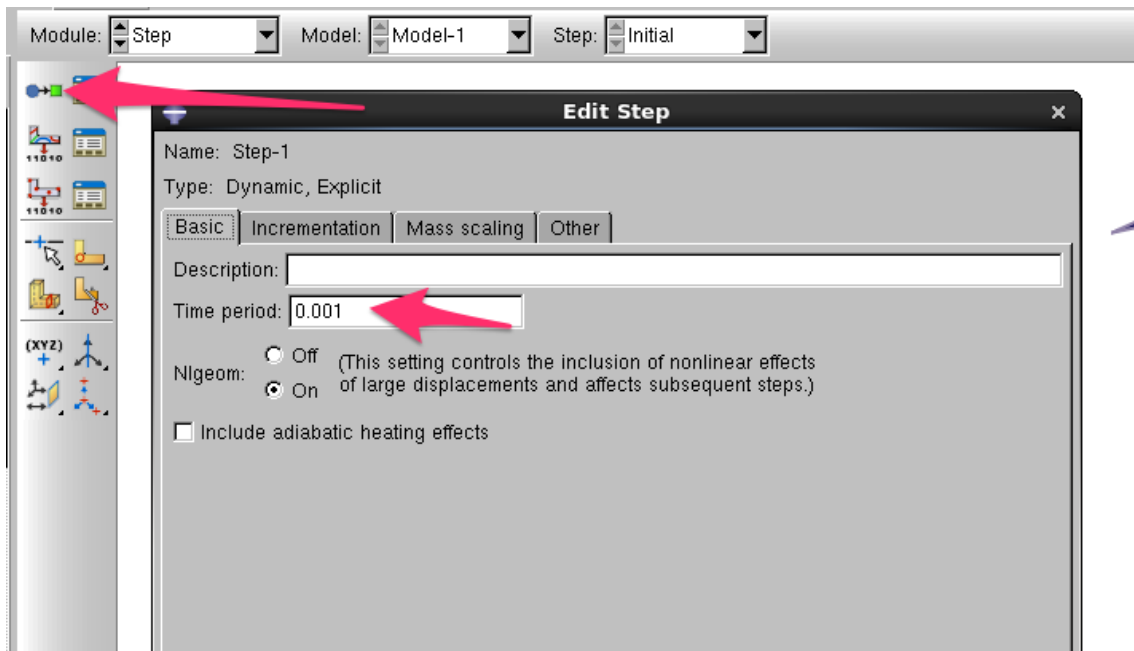
4. Create an assembly

Instance the parts in the assembly module.



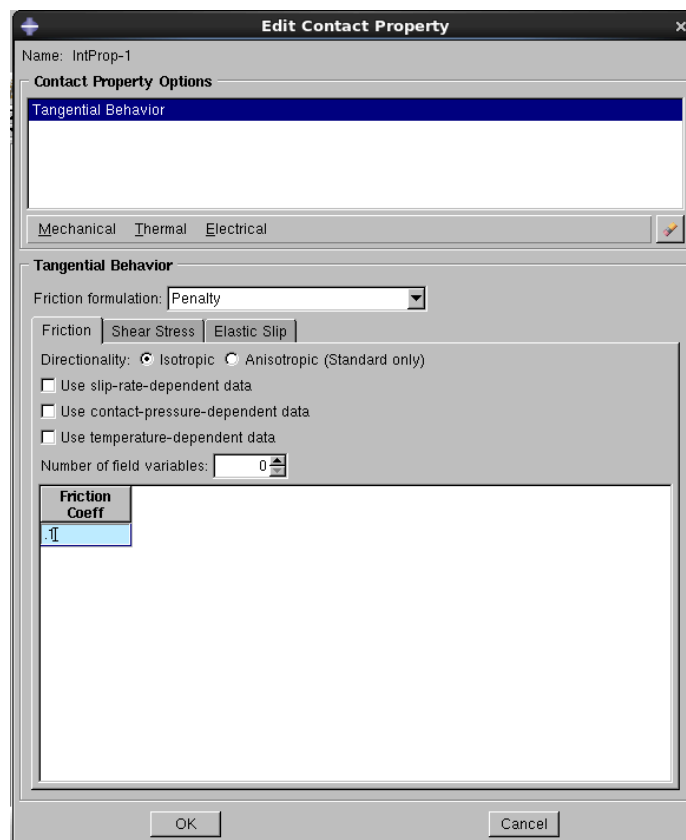
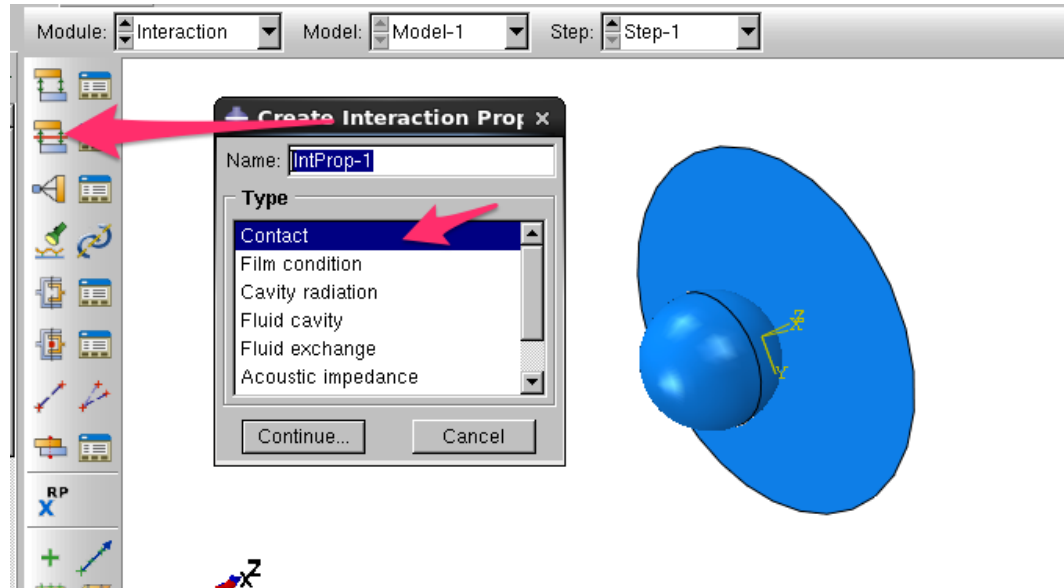
5. Define the analysis step

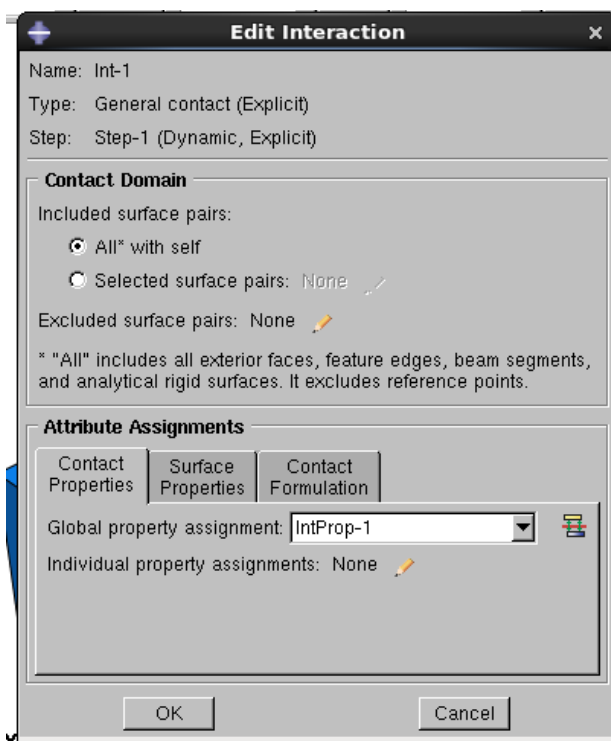
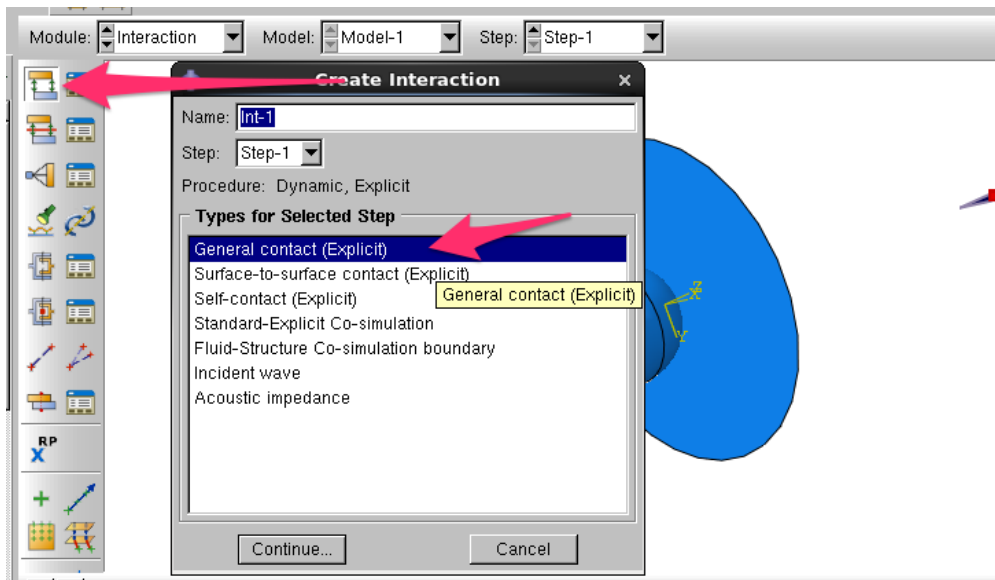
Create a dynamic explicit step and include Nlgeom. Set the time period = 0.001 s



6. Define the Interactions (contact conditions)

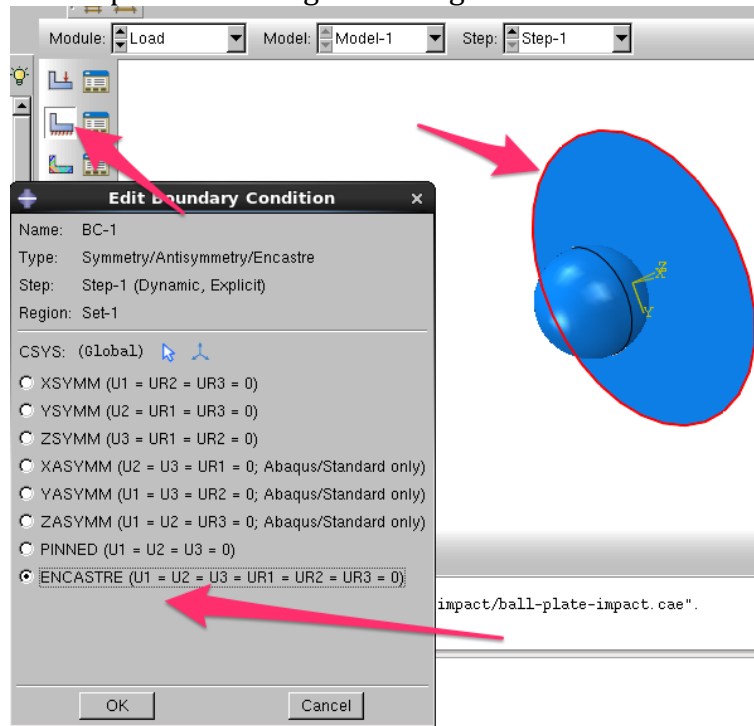
The “catch all” general contact with the default All* with self option provides all the contact settings necessary. In this case a global interaction property defining a coefficient of friction of 0.1 is used.



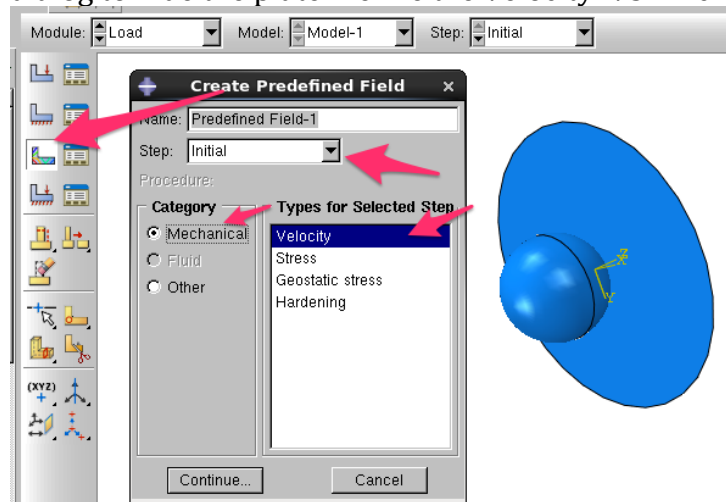



7. Define loads and restraints

Fix the plate at the edge in all degrees of freedom.



The ball is given an initial velocity, predefined field > mechanical > velocity. Ensure the entire ball is selected and not just the outer surface, this is best done using the assembly display options dialog to hide the plate. Define the Velocity: $V3 = 40\text{m/s}$.



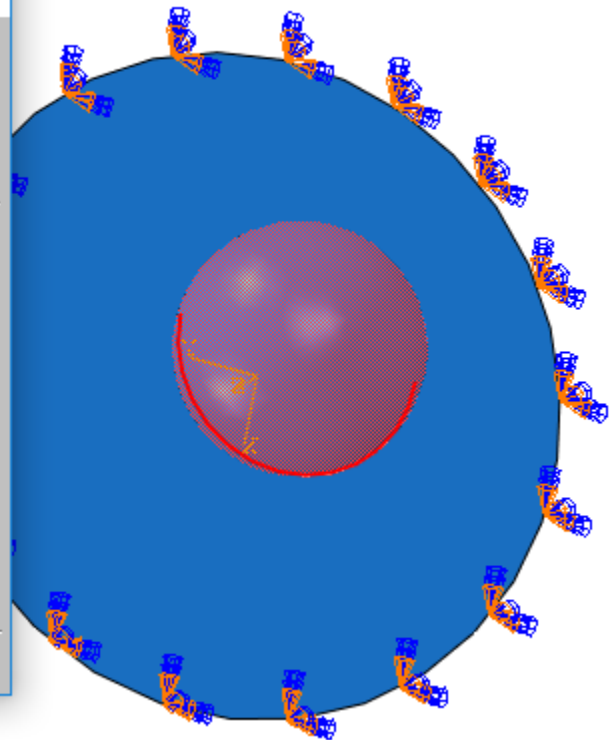

Edit Predefined Field
×

Name: Predefined Field-1
Type: Velocity
Step: Initial
Region: Set-2

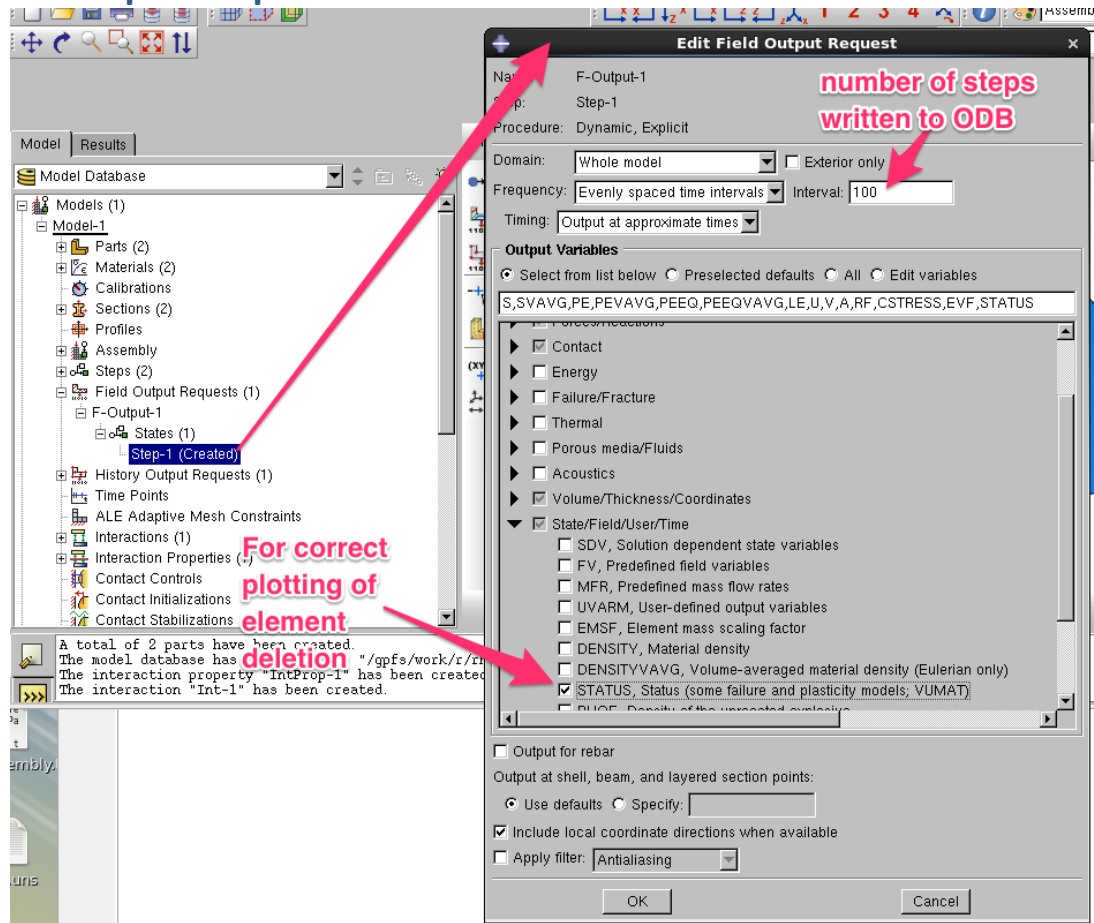
Distribution: Uniform f(x)
Definition: Translational only

V1:
V2:
V3:

Angular velocity: radians/time
Axis point 1:
Axis point 2:

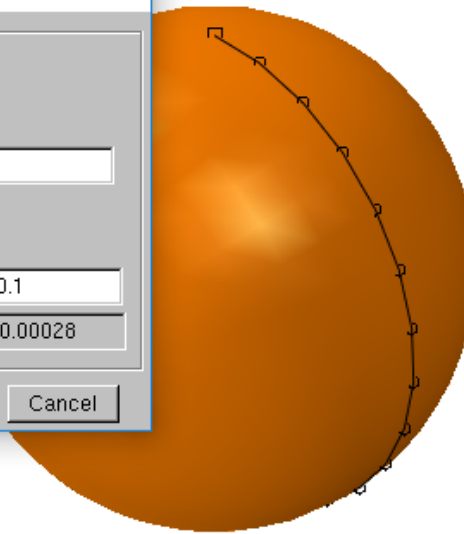
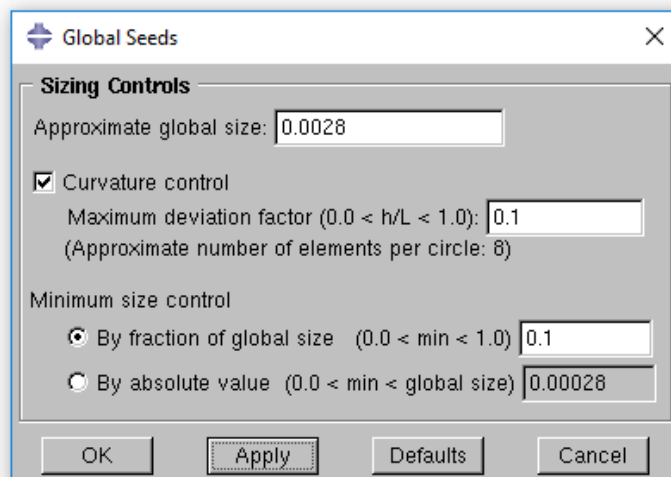


8. Output Requests

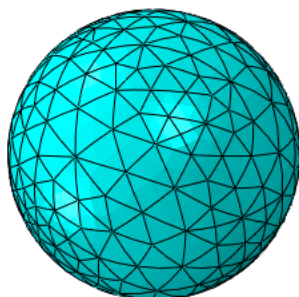
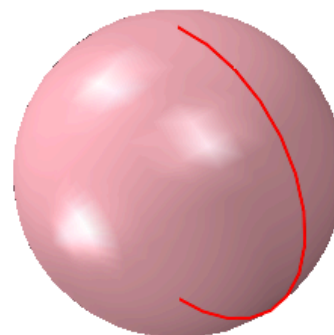
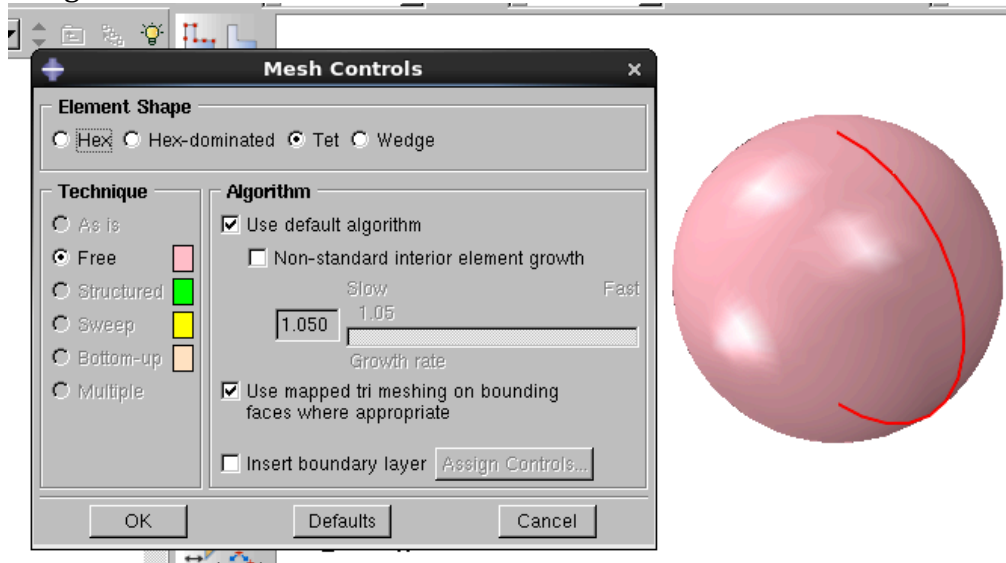


9. Create the Mesh

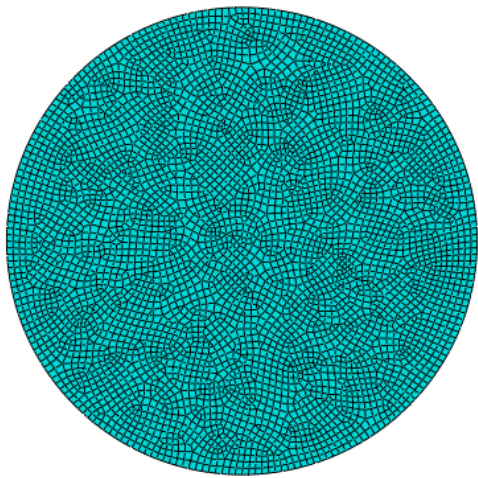
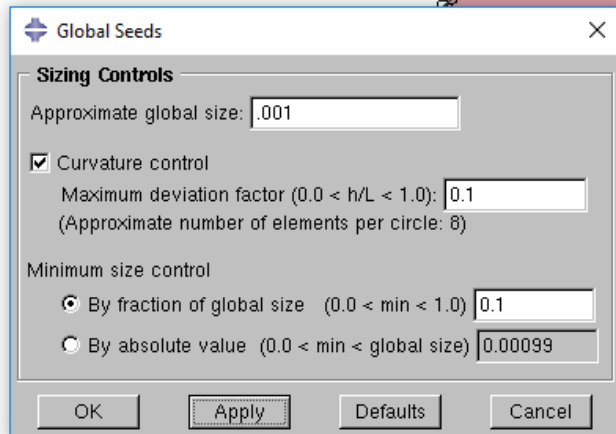
Mesh the ball. Use seed of .0028



Assign tet mesh for ball:

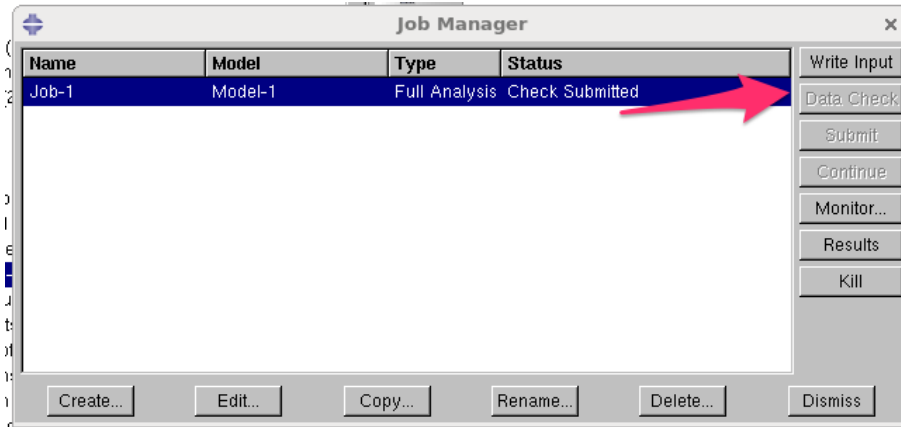


Use a mesh seed size of 0.001 for the plate:

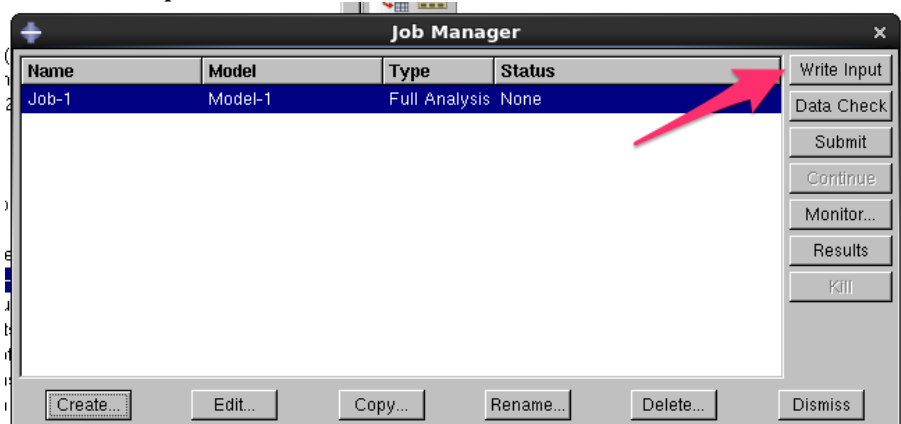


10. Create a job and then submit it using PBS

Submit the job and run a data check:



write the input file:



Open a terminal and ssh to aci-b. Navigate to your directory. Submit pbs script:

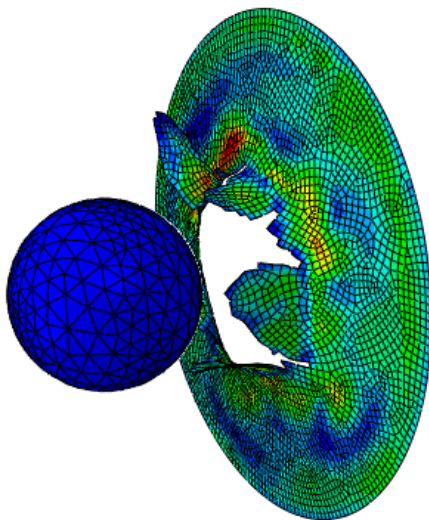
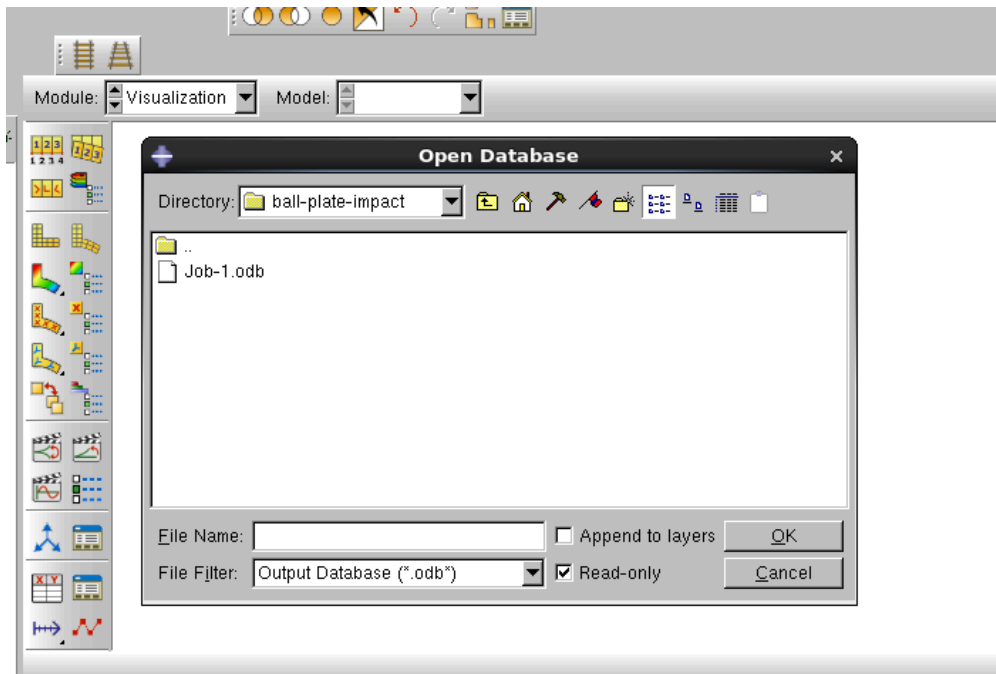
```
-rwxrwx--- 1 rhk12 rhk12_collab 917 Feb 7 14:55 Job-1.log
-rw-rw---- 1 rhk12 rhk12_collab 917508 Feb 7 14:57 Job-1.inp
-rw-rw---- 1 rhk12 rhk12_collab 344064 Feb 7 15:14 app4_v1.cae
-rw-rw---- 1 rhk12 rhk12_collab 10341 Feb 7 15:14 abaqus.rpy
-rw-rw---- 1 rhk12 rhk12_collab 10734 Feb 7 15:14 app4_v1.jnl
-rw-r----- 1 rhk12 rhk12_collab 281 Feb 7 15:24 abaqus.pbs
[rhk12@aci-lgn-007 ball-impact]$ qsub -A open abaqus.pbs
```

Monitor the job:

```
[rhk12@aci-lgn-007 ball-impact]$ qstat -u rhk12
torque01.util.production.int.aci.ics.psu.edu:
Job ID          Username      Queue    Jobname     SessID  NDS   TSK   Req'd    Req'd    Elap
-----
10536102.torque01.util rhk12      open     abaqus.pbs  --      1     2      4gb    00:30:00 Q      --
[rhk12@aci-lgn-007 ball-impact]$
```

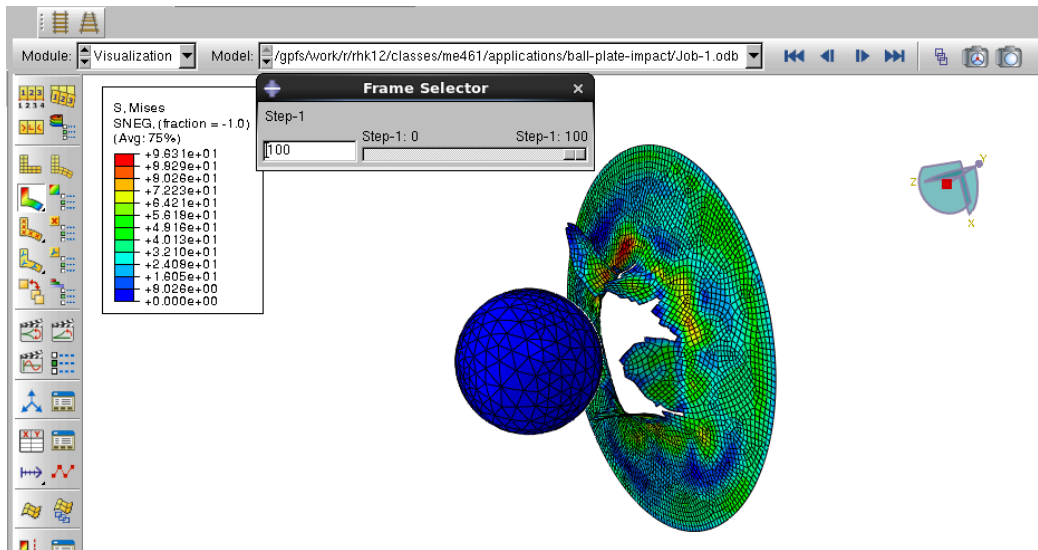
11. Open the results file and plot the results

Once the job is complete, go to the visualization module and open the ODB file:



baqus/Explicit 6.14-2 Tue Mar 01 14:59:40 EST 2016

Use the frame selector to quickly move through the time steps:



In the undeformed configuration you can see the fully damaged/ deleted elements:

