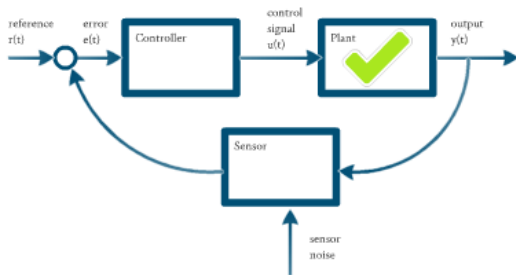


## Closed Loop Control System (recap)



## PID Controller Design



$$u(t) = \underbrace{K_p e(t)}_{\text{(P)roportional control}} + \underbrace{K_i \int_0^t e(\tau) d\tau}_{\text{(I)ntegral control}} + \underbrace{K_d \frac{d}{dt} e(t)}_{\text{(D)ifferential control}}$$

Component	Error Type			
	Stability	Fast transient response	Zero steady-state error	Small overshoot
Proportional	Stable	Fast	Non-zero	Large
Integral	Stable	Slow	Zero	Large
Differential	Stable	Fast	Non-zero	Small

## Application of PID Controllers

### First Order System

$$P(s) = \frac{A}{s + B}$$

Proportional Control  $K_{cp} = K_c$

$$\frac{Y(s)}{U(s)} = \frac{A}{s + B + K_c A}$$

$$\frac{Y(s)}{U(s)} = \frac{A}{s + \frac{B + K_c A}{A}}$$

$$\frac{Y(s)}{U(s)} = \frac{A}{s + \frac{B}{A} + K_c}$$

$$\frac{Y(s)}{U(s)} = \frac{A}{s + \frac{B}{A} + K_c}$$

### Second Order System

$$P(s) = \frac{b_2 s + b_1}{s^2 + a_1 s + a_0}$$

$$\frac{Y(s)}{U(s)} = \frac{b_2 s + b_1}{s^2 + a_1 s + a_0 + K_c (b_2 s + b_1)}$$

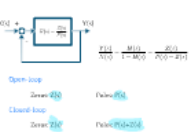
$$\frac{Y(s)}{U(s)} = \frac{b_2 s + b_1}{s^2 + (a_1 + K_c b_2) s + (a_0 + K_c b_1)}$$

$$\frac{Y(s)}{U(s)} = \frac{b_2 s + b_1}{s^2 + \zeta \omega_n s + \omega_n^2}$$

$$\zeta = \frac{a_1 + K_c b_2}{2 \omega_n}$$

$$\omega_n^2 = a_0 + K_c b_1$$

### Finding the Open and Closed Loop Poles and Zeros



## ELEC 207 Part B

### Control Theory Lecture 7: Simple Control System Design

Prof Simon Maskell  
CHAD-G68  
s.maskell@liverpool.ac.uk  
0151 794 4573



This lecture covers:

- Three term PID controller
- Finding the open and closed loop poles and zeros
- Pole and zero placement design for a PID controller

# ELEC 207 Part B

## Control Theory Lecture 7: Simple Control System Design

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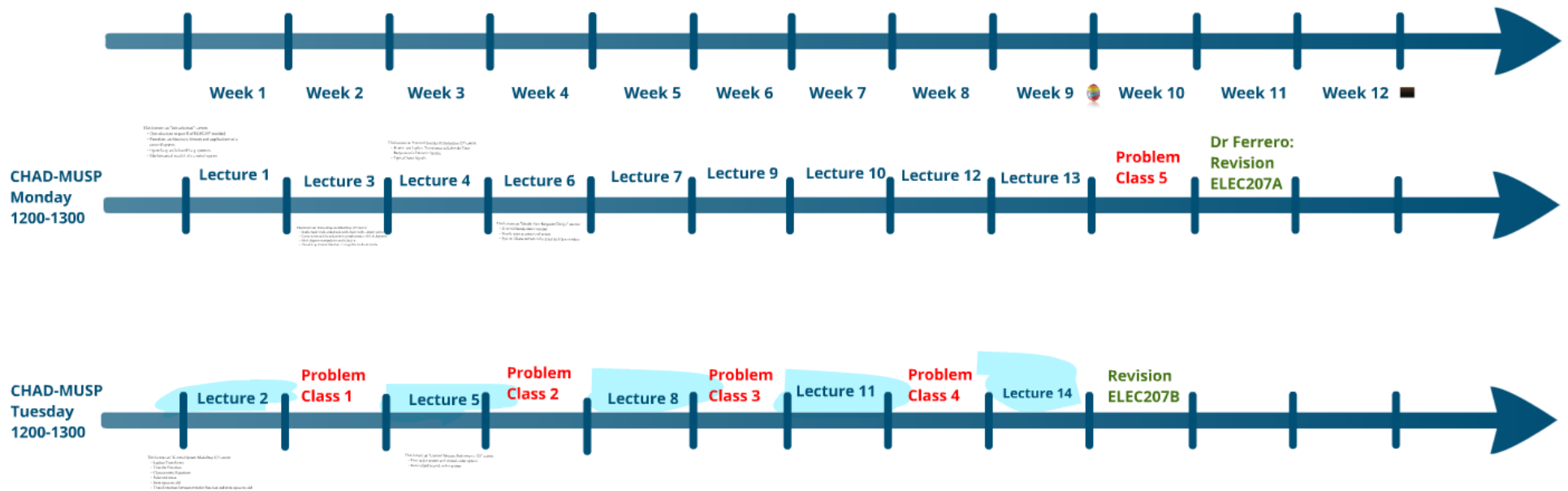
UNIVERSITY OF  
LIVERPOOL

This lecture covers:

- Three term PID controller
- Finding the open and closed loop poles and zeros
- Pole and zero placement design for a PID controller



# ELEC 207B: Timeline



This lecture on "Introduction" covers:

- (Introduction to part B of ELEC207 module)
- Function, architecture, history and applications of a control system
- Open-loop and closed-loop systems
- Mathematical model of a control system

# Lecture 1

A decorative graphic consisting of a thick horizontal blue bar and a thick vertical blue bar that intersect to form a cross-like shape, positioned at the bottom of the slide.

# Lecture 2

This lecture on "Control System Modelling (1)" covers:

- Laplace Transforms
- Transfer Function
- Characteristic Equations
- Poles and zeros
- State-space model
- Transformation between transfer function and state-space model

1

# Lecture 3

Lecture

This lecture on "Control Systems B

- How to use Laplace Transform
- Response of a Dynamic System
- Typical Input Signals

This lecture on "Control System Modelling (2)" covers:

- Single-input single-output and multi-input multi-output systems
- Components and the underpinning mathematics of block diagrams
- Block diagram manipulation and reduction
- Closed-loop transfer function of a negative feedback system

This lecture on "Control Systems Performance (1)" covers:

- How to use Laplace Transforms to Solve the Time Response of a Dynamic System
- Typical Input Signals

# Lecture 4

Le



# Lecture 5

This lecture on "Control Systems Performance (2)" covers:

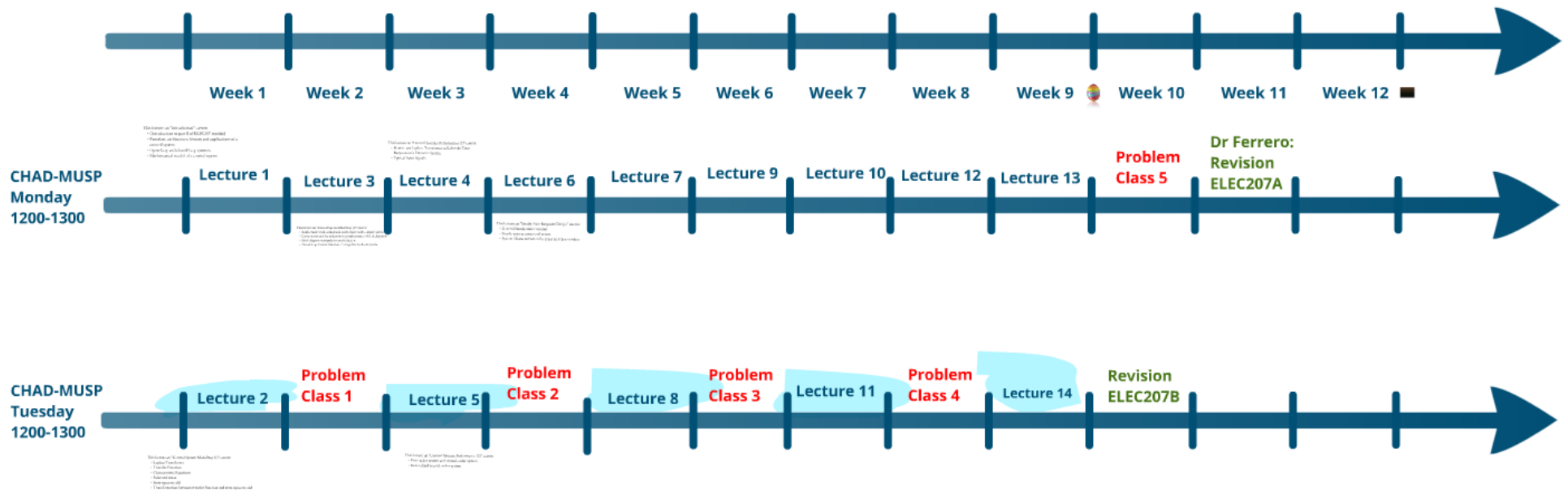
- First-order system and second-order system
- Generalized second-order system

# Lecture 6

This lecture on "Steady-State Response Design" covers:

- General Steady-state response
- Steady-state accuracy and errors
- System Characterization by order and type number

# ELEC 207B: Timeline

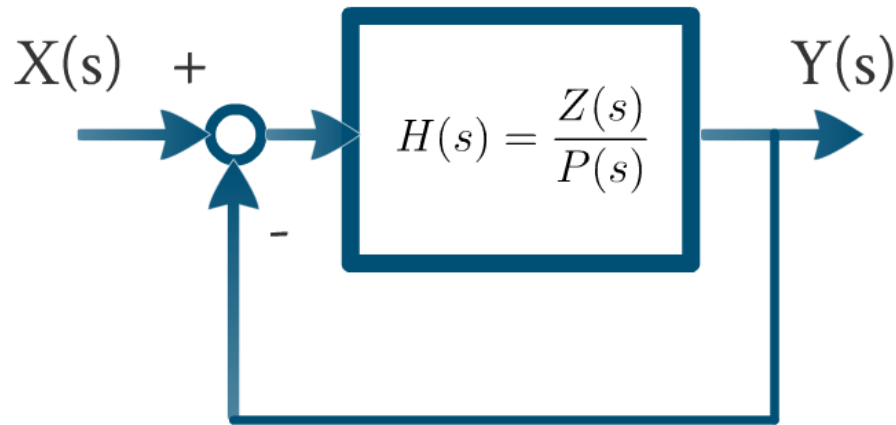


This lecture covers:

- Three term PID controller
- Finding the open and closed loop poles and zeros
- Pole and zero placement design for a PID controller



# Finding the Open and Closed Loop Poles and Zeros



$$\frac{Y(s)}{X(s)} = \frac{H(s)}{1 + H(s)} = \frac{Z(s)}{P(s) + Z(s)}$$

Open-loop

Zeros:  $Z(s)$

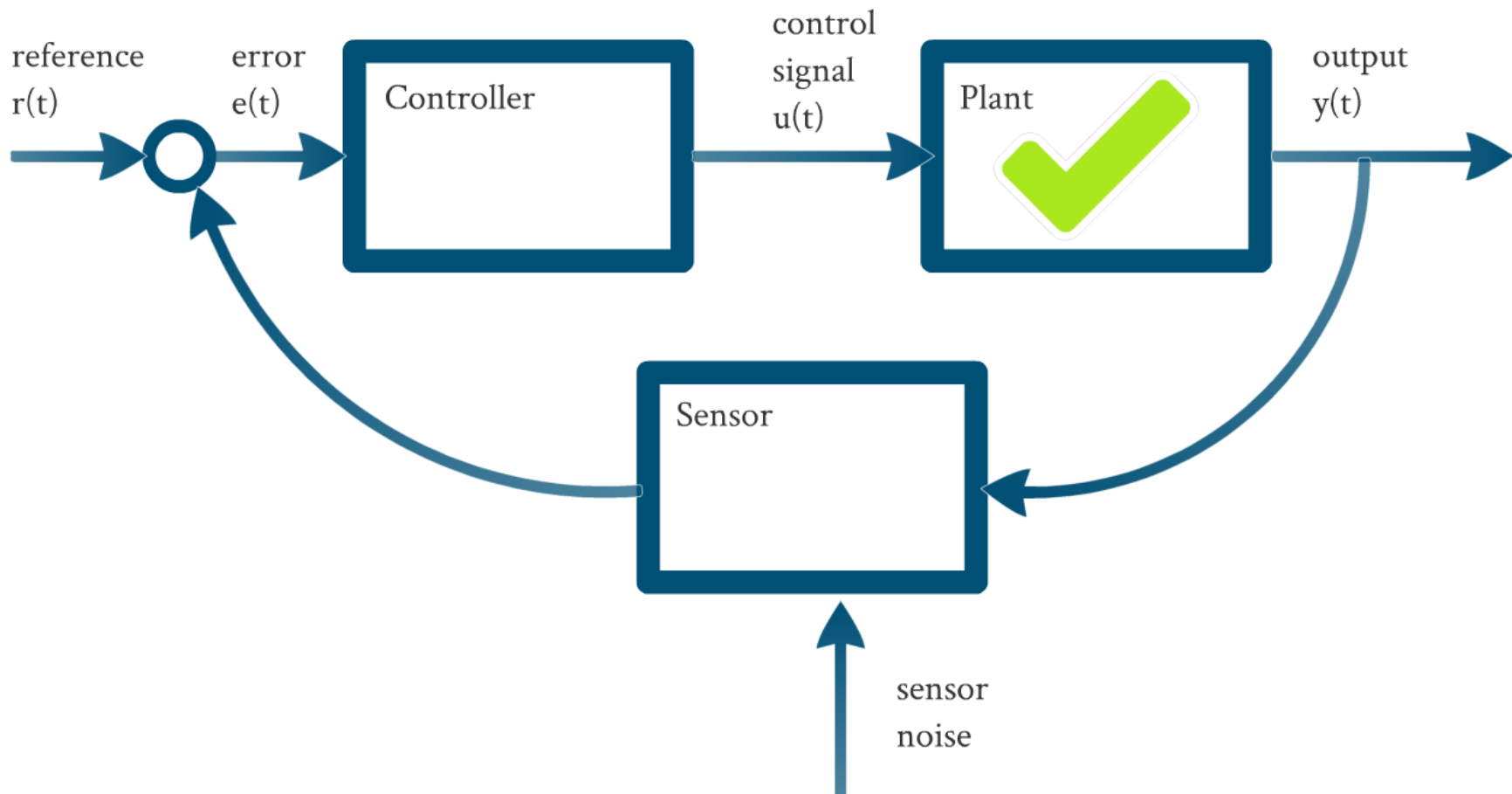
Poles:  $P(s)$

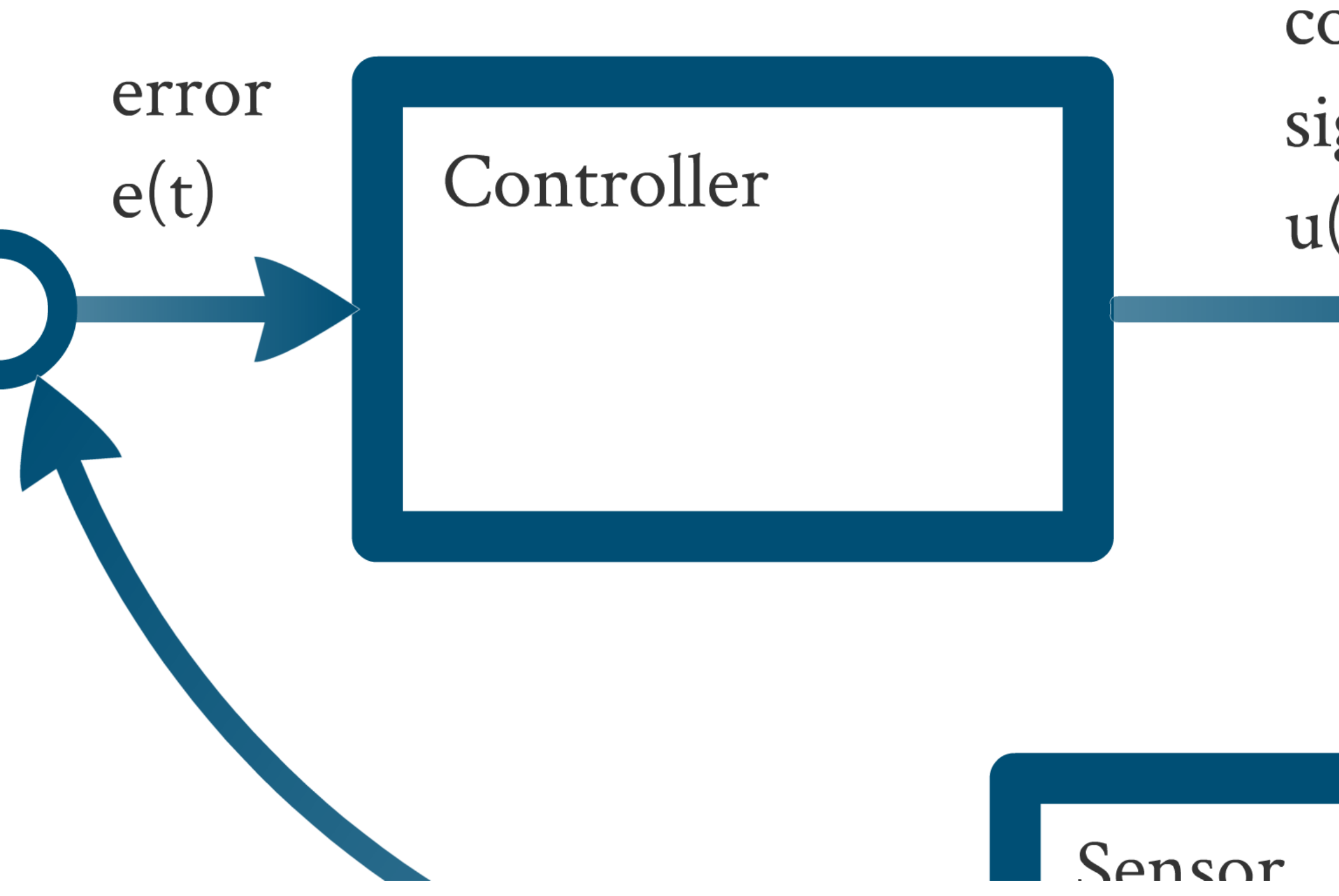
Closed-loop

Zeros:  $Z(s)$

Poles:  $P(s) + Z(s)$

# Closed Loop Control System (recap)





# PID Controller Design





























$$u(t) = \underbrace{K_p e(t)}_{\text{(P)roportional control}} + \underbrace{K_i \int_0^t e(\tau) d\tau}_{\text{(I)ntegral control}} + \underbrace{K_d \frac{d}{dt} e(t)}_{\text{(D)ifferential control}}$$

Component	Error Type			
	Stability	Fast transient response	Zero steady-state error	Small overshoot
Proportional	Excessively large values cause instability	Large values slow the response	Non-zero steady-state errors can result	Large overshoot will cause
Integral	Integral control helps stability	The transient is compensated for with less overshoot	Integral control gives the steady-state error to zero	Any overshoot is minimized for with overshoot
Differential	Excessively large values cause instability	Differential control slows response down to	Non-zero steady-state error result	Decreases overshoot



$$u(t) = \underbrace{K_p e(t)}_{\text{(P)roportional control}} + \underbrace{K_i \int_0^t e(\tau) d\tau}_{\text{(I)ntegral control}} + \underbrace{K_d \frac{d}{dt} e(t)}_{\text{(D)ifferential control}}$$

Component	Error Type			
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Proportional	Excessively large  cause instability	Large values  the response	Non-zero steady  errors can result	Large  will cause  shoot
Integral	Integral  lps stability	The trans  s to  compensated for with the  nsult  overshoot	Integral  gives the steady-s  r to zero	Any  er  ited for with overshoot
Differential	Excess  y low  values amp  s  cause instability	Diffe  c  slows down the  ns  response	Non-ze  dy-state errors  result	 es decre  rshoot

# Stability

Excessively large  cause instability

Integral  instability

# response

---

Large values



the response

The trans is to compensated  
for with the consum overshoot



# steady-state error

---

Non-zero steady-state errors can result



Integral action drives the steady-state error to zero


























# OVERSHOOT

Large  will  
cause overshoot

Any  error  
transient  be  ated  
for  overshoot

$$u(t) = \underbrace{K_p e(t)}_{\text{(P)roportional control}} + \underbrace{K_i \int_0^t e(\tau) d\tau}_{\text{(I)ntegral control}} + \underbrace{K_d \frac{d}{dt} e(t)}_{\text{(D)ifferential control}}$$

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Excessively large  cause instability



Integral  helps stability



Excessively large values



Large values



the response

The trans to compensated  
for with the consumer overshoot



Difference slows  
down the response







---

Non-zero steady-state errors can result



Integral drives the steady-state error to zero



Non-zero steady-state errors can result



Larger  $\zeta$  will  
cause overshoot





























Any  $\zeta < 1$  er  
transient response be damped  
for with overshoot





1.  $\zeta > 1$  es  
decreases overshoot



$$u(t) = \underbrace{K_p e(t)}_{\text{(P)roportional control}} + \underbrace{K_i \int_0^t e(\tau) d\tau}_{\text{(I)ntegral control}} + \underbrace{K_d \frac{d}{dt} e(t)}_{\text{(D)ifferential control}}$$

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Integral cips stability

Excessive ly la values  
ampsause  
instability

for with the consumer overshoot



Difference slows  
down the consumer response



Integral drives the  
steady-state error to zero






























Non-zero steady-state  
errors are a result



for with overshoot



$$u(t) = \underbrace{K_p e(t)}_{\text{(P)roportional control}} + \underbrace{K_i \int_0^t e(\tau) d\tau}_{\text{(I)ntegral control}} + \underbrace{K_d \frac{d}{dt} e(t)}_{\text{(D)ifferential control}}$$

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# Application of PID Controllers

## First Order System

$$P(s) = \frac{b}{s + a}$$

Proportional Control

$$C(s) = K_p$$

$$\frac{Y(s)}{X(s)} = \frac{P(s)C(s)}{1 + P(s)C(s)}$$

$$= \frac{K_p \frac{b}{s+a}}{1 + K_p \frac{b}{s+a}}$$

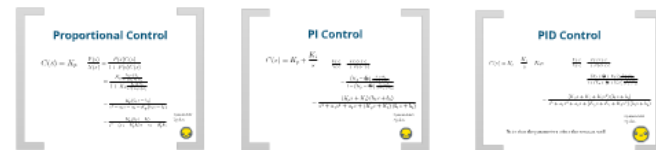
$$= \frac{K_p b}{s + a + K_p b}$$

1 parameter  
1 pole



## Second Order System

$$P(s) = \frac{b_1 s + b_2}{s^2 + a_1 s + a_2}$$





You **Tube**



You **Tube**

# Application of PID Controllers

## First Order System

$$P(s) = \frac{b}{s + a}$$

Proportional Control

$$C(s) = K_p$$

$$\frac{Y(s)}{X(s)} = \frac{P(s)C(s)}{1 + P(s)C(s)}$$

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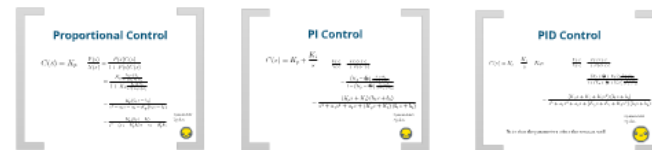
$$= \frac{K_p b}{s + a + K_p b}$$

1 parameter  
1 pole



## Second Order System

$$P(s) = \frac{b_1 s + b_2}{s^2 + a_1 s + a_2}$$



# First Order System

Proportional Control

$$P(s) = \frac{b}{s + a}$$

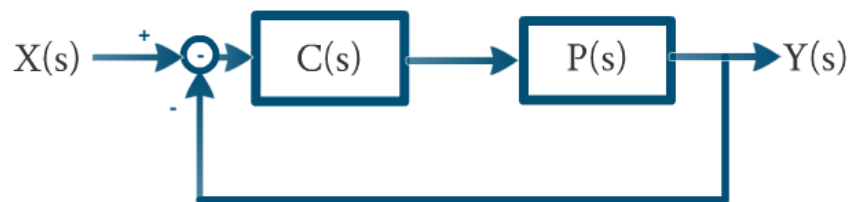
$$C(s) = K_p$$

$$\frac{Y(s)}{X(s)} = \frac{P(s)C(s)}{1 + P(s)C(s)}$$

$$= \frac{K_p \frac{b}{s+a}}{1 + K_p \frac{b}{s+a}}$$

$$= \frac{K_p b}{s + a + K_p b}$$

1 parameter  
1 pole



# Second Order System

$$P(s) = \frac{b_1 s + b_2}{s^2 + a_1 s + a_2}$$

## Proportional Control

$$\begin{aligned} C(s) = K_p \quad \frac{Y(s)}{X(s)} &= \frac{P(s)C(s)}{1 + P(s)C(s)} \\ &= \frac{K_p \frac{b_1 s + b_2}{s^2 + a_1 s + a_2}}{1 + K_p \frac{b_1 s + b_2}{s^2 + a_1 s + a_2}} \\ &= \frac{K_p (b_1 s + b_2)}{s^2 + a_1 s + a_2 + K_p (b_1 s + b_2)} \\ &= \frac{K_p (b_1 s + b_2)}{s^2 + (a_1 + K_p b_1) s + a_2 + K_p b_2} \end{aligned}$$

1 parameter  
2 poles



## PI Control

$$\begin{aligned} C(s) = K_p + \frac{K_i}{s} \quad \frac{Y(s)}{X(s)} &= \frac{P(s)C(s)}{1 + P(s)C(s)} \\ &= \frac{(K_p + \frac{K_i}{s}) \frac{b_1 s + b_2}{s^2 + a_1 s + a_2}}{1 + (K_p + \frac{K_i}{s}) \frac{b_1 s + b_2}{s^2 + a_1 s + a_2}} \\ &= \frac{(K_p s + K_i) (b_1 s + b_2)}{s^3 + a_1 s^2 + a_2 s + (K_p s + K_i) (b_1 s + b_2)} \end{aligned}$$

2 parameters  
3 poles



## PID Control

$$\begin{aligned} C(s) = K_p + \frac{K_i}{s} + K_d s \quad \frac{Y(s)}{X(s)} &= \frac{P(s)C(s)}{1 + P(s)C(s)} \\ &= \frac{(K_p + \frac{K_i}{s} + K_d s) \frac{b_1 s + b_2}{s^2 + a_1 s + a_2}}{1 + (K_p + \frac{K_i}{s} + K_d s) \frac{b_1 s + b_2}{s^2 + a_1 s + a_2}} \\ &= \frac{(K_p s^2 + K_i s + K_d s^2) (b_1 s + b_2)}{s^3 + a_1 s^2 + a_2 s + (K_p s^2 + K_i s + K_d s^2) (b_1 s + b_2)} \end{aligned}$$

3 parameters  
3 poles

Note that the parameters affect the zeros as well



# Proportional Control

$$\begin{aligned} C(s) &= K_p \quad \frac{Y(s)}{X(s)} = \frac{P(s)C(s)}{1 + P(s)C(s)} \\ &= \frac{K_p \frac{b_1 s + b_2}{s^2 + a_1 s + a_2}}{1 + K_p \frac{b_1 s + b_2}{s^2 + a_1 s + a_2}} \\ &= \frac{K_p (b_1 s + b_2)}{s^2 + a_1 s + a_2 + K_p (b_1 s + b_2)} \\ &= \frac{K_p (b_1 s + b_2)}{s^2 + (a_1 + K_p b_1) s + a_2 + K_p b_2} \end{aligned}$$

1 parameter

2 poles



# PI Control

$$C(s) = K_p + \frac{K_i}{s}$$

$$\frac{Y(s)}{X(s)} = \frac{P(s)C(s)}{1 + P(s)C(s)}$$

$$= \frac{\left(K_p + \frac{K_i}{s}\right) \frac{b_1 s + b_2}{s^2 + a_1 s + a_2}}{1 + \left(K_p + \frac{K_i}{s}\right) \frac{b_1 s + b_2}{s^2 + a_1 s + a_2}}$$

$$= \frac{(K_p s + K_i)(b_1 s + b_2)}{s^3 + a_1 s^2 + a_2 s + (K_p s + K_i)(b_1 s + b_2)}$$

2 parameters

3 poles





# PID Control

$$C(s) = K_p + \frac{K_i}{s} + K_d s$$

$$\frac{Y(s)}{X(s)} = \frac{P(s)C(s)}{1 + P(s)C(s)}$$

$$= \frac{(K_p + \frac{K_i}{s} + K_d s) \frac{b_1 s + b_2}{s^2 + a_1 s + a_2}}{1 + (K_p + \frac{K_i}{s} + K_d s) \frac{b_1 s + b_2}{s^2 + a_1 s + a_2}}$$

$$= \frac{(K_p s + K_i + K_d s^2)(b_1 s + b_2)}{s^3 + a_1 s^2 + a_2 s + (K_p s + K_i + K_d s^2)(b_1 s + b_2)}$$

3 parameters

3 poles

Note that the parameters affect the zeros as well



# Second Order System

$$P(s) = \frac{b_1 s + b_2}{s^2 + a_1 s + a_2}$$

## Proportional Control

$$\begin{aligned} C(s) = K_p \quad \frac{Y(s)}{X(s)} &= \frac{P(s)C(s)}{1 + P(s)C(s)} \\ &= \frac{K_p \frac{b_1 s + b_2}{s^2 + a_1 s + a_2}}{1 + K_p \frac{b_1 s + b_2}{s^2 + a_1 s + a_2}} \\ &= \frac{K_p (b_1 s + b_2)}{s^2 + a_1 s + a_2 + K_p (b_1 s + b_2)} \\ &= \frac{K_p (b_1 s + b_2)}{s^2 + (a_1 + K_p b_1) s + a_2 + K_p b_2} \end{aligned}$$

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2 parameters  
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## PID Control

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3 parameters  
3 poles



Note that the parameters affect the zeros as well

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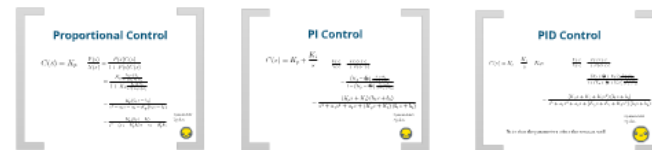
$$= \frac{K_p b}{s + a + K_p b}$$

1 parameter  
1 pole



## Second Order System

$$P(s) = \frac{b_1 s + b_2}{s^2 + a_1 s + a_2}$$



This lecture covers:

- Three term PID controller
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- Pole and zero placement design for a PID controller

