

24-650 Applied Finite Element Analysis

Assignment 7

submitted by

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Objective

The goal of this assignment is to perform an analysis of a masonry chimney. The results are:

Maximum Displacement: 1368mm

Safety Factor = 0.065783, so it will **collapse** since much less than 1.

Assumptions and Loading Conditions

- 1) The masonry chimney is made of concrete material: $E=30,000$ MPa, Poisson's ratio = 0.18, Ultimate Tensile Strength = 5 MPa.
- 2) Assume the pressure wave impacts exactly 2 surfaces on one side of the chimney.
- 3) The pressure wave is indicated below:

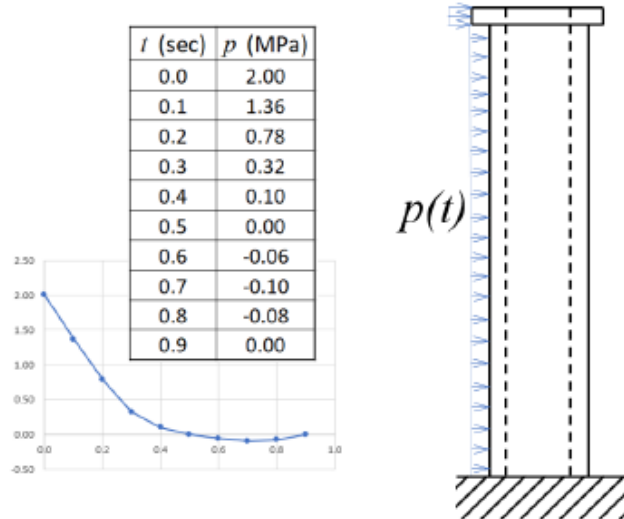


Figure 1. Pressure distribution.

Model

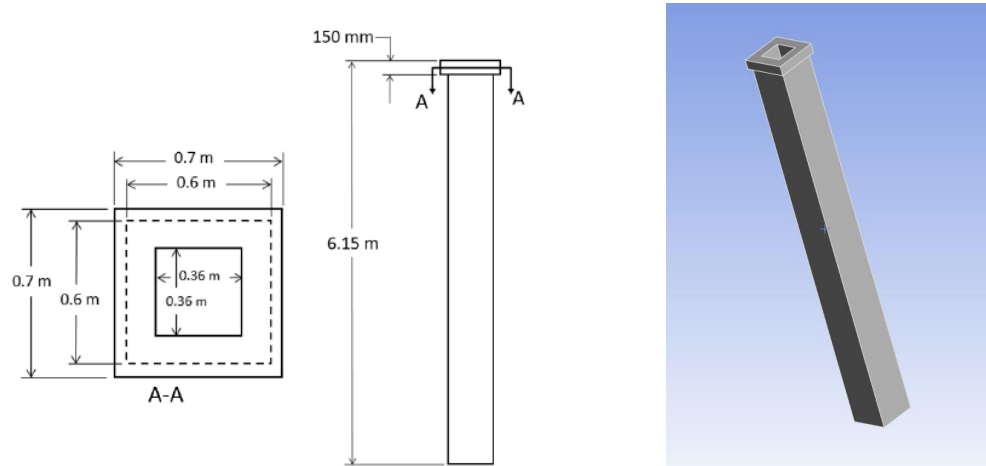


Figure 2. The geometry of the chimney.

Boundary Conditions

The boundary conditions of support on bottom of the pin is indicated below.

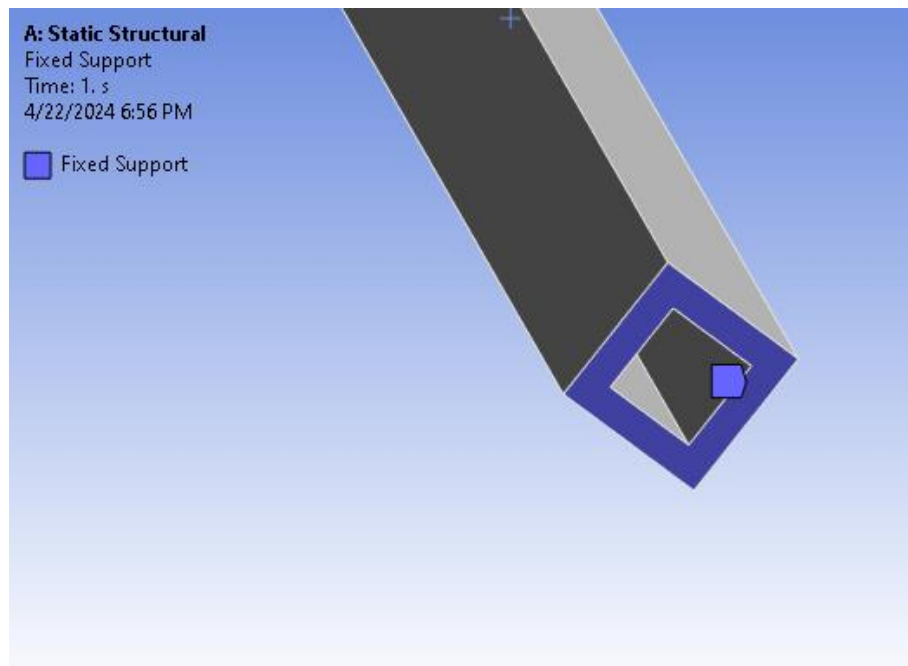


Figure 3. Fixed Support BC

Modal analysis - Analysis Settings

For model analysis, the maximum of modes has been selected to **22**, since mode of 20 can guarantee the ratio of effective mass is above 90%, with some extra safety margin of 2.

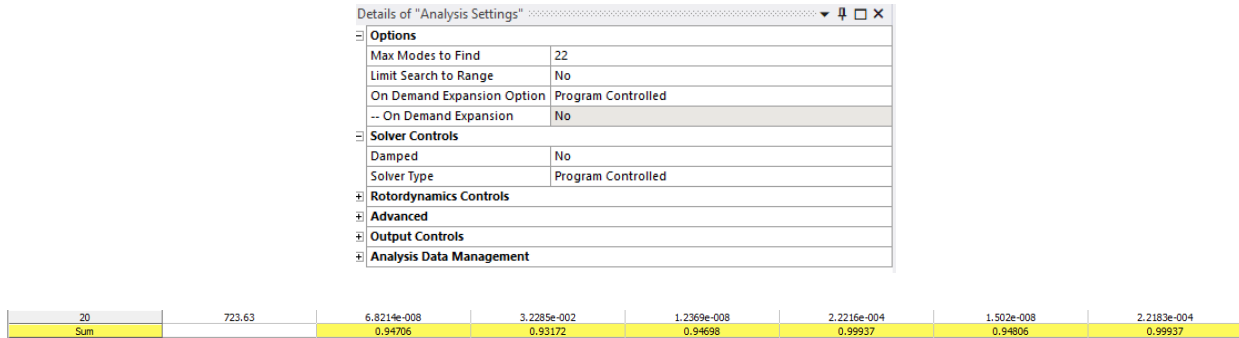


Figure 4. Modal analysis settings

(As indicated above, modal of 20 is capable to maintain a ratio with minimum of 93%, so modal of **22** is selected to add some extra margin)

Harmonic analysis - Analysis Settings

For model analysis, the pressure force has been applied (see in figure below):

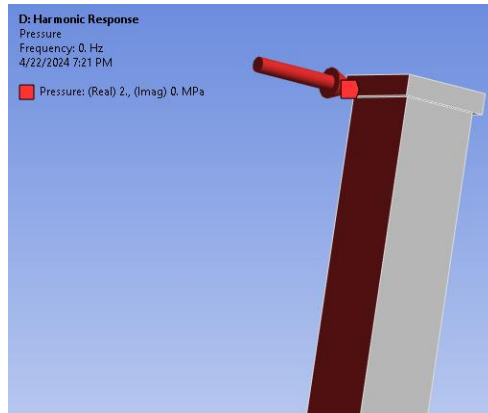


Figure 5. Pressure force.

And the maximum frequency range has set to 300Hz, to include any frequency in the first 10 mode from modal analysis, with the damping set to indicated value (3%):

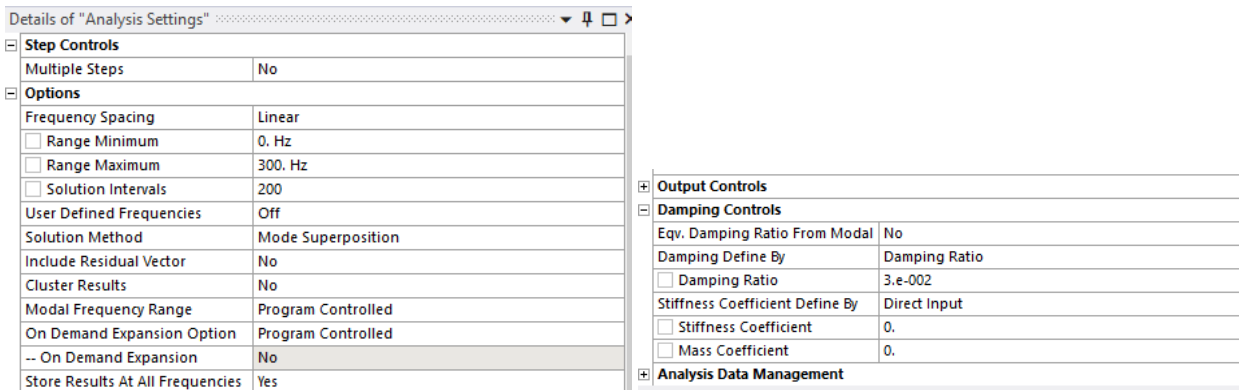


Figure 6. Harmonic analysis settings.

Transient structural analysis - Analysis Settings

For model analysis, the pressure force has been applied, with tabular data (see in figure below):

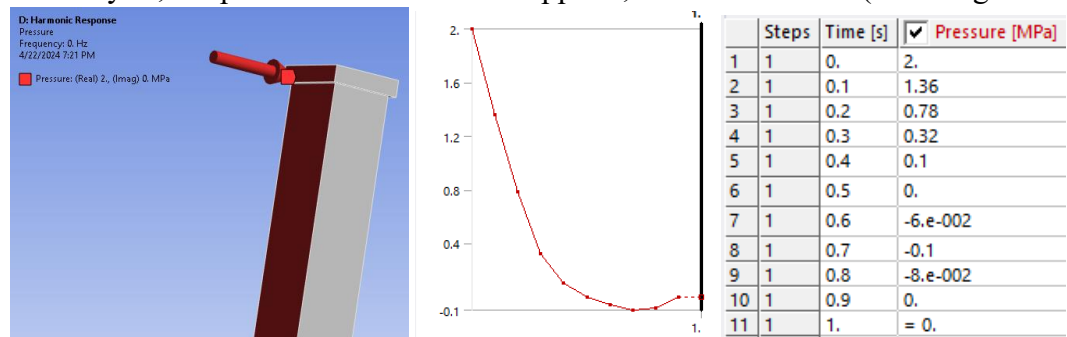


Figure 7. Pressure force.

Step Controls		Damping Controls	
Number Of Steps	1.	Eqv. Damping Ratio From Modal	No
Current Step Number	1.	<input type="checkbox"/> Damping Ratio	0.
Step End Time	1. s	Stiffness Coefficient Define By	Damping vs Frequency
Auto Time Stepping	Off	-- Frequency	10. Hz
Define By	Time	-- Damping Ratio	3.e-002
Time Step	1.e-003 s	Stiffness Coefficient	9.5493e-004
Time Integration	On	<input type="checkbox"/> Mass Coefficient	0.
		Numerical Damping	Program Controlled
		Numerical Damping Value	0.005

Figure 8. Transient analysis settings.

For the time step controls, the final value of 1e-3 has been applied after convergence study, the convergence study will be introduced in the results section. As for the damping controls, the damping ratio of **3% @ 10.4Hz** is selected from the first mode in the modal analysis. Also, the safety factor will be investigated, and the configuration is indicated below:

Definition	
Theory	Max Tensile Stress
Stress Limit Type	Tensile Ultimate Per Material

Figure 9. Safety factor configuration.

Mech Settings and Solution Convergence

The mesh setting **stays at default** in this project.

For frequency settings, the default value (**frequency intervals = 300**) gives the curve below, as one can see the curve is not smooth, and after the trading-off between computing time and convergence requirement, the **frequency intervals = 300** gives the most balanced result (converged to 4.52-4.73mm):

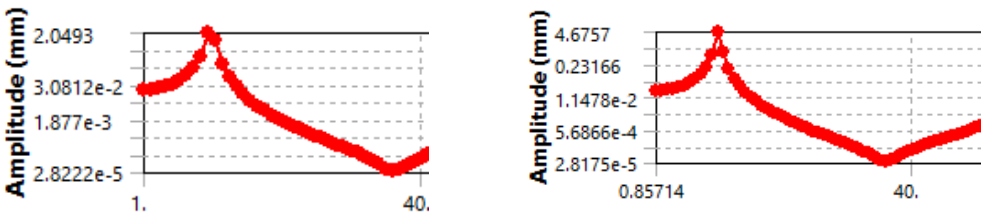


Figure 10. Freq. No. = 300 (left) Freq. No. = 350 (right), x-direction Amp.

For the time step settings, the default value (**time step = 0.1**) gives the curve below, as one can see the curve is not smooth, and after the trading-off between computing time and convergence requirement, the time step of **1e-3** gives the most balanced result (converged to 1375-1395mm):

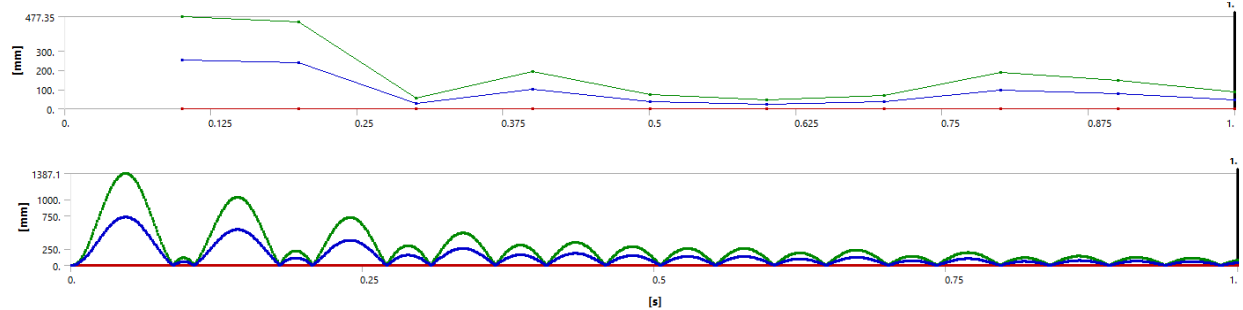


Figure 11. Time step = 0.1s (above) Time step = 1e-3s (below), total deformation.

Results (Part A)

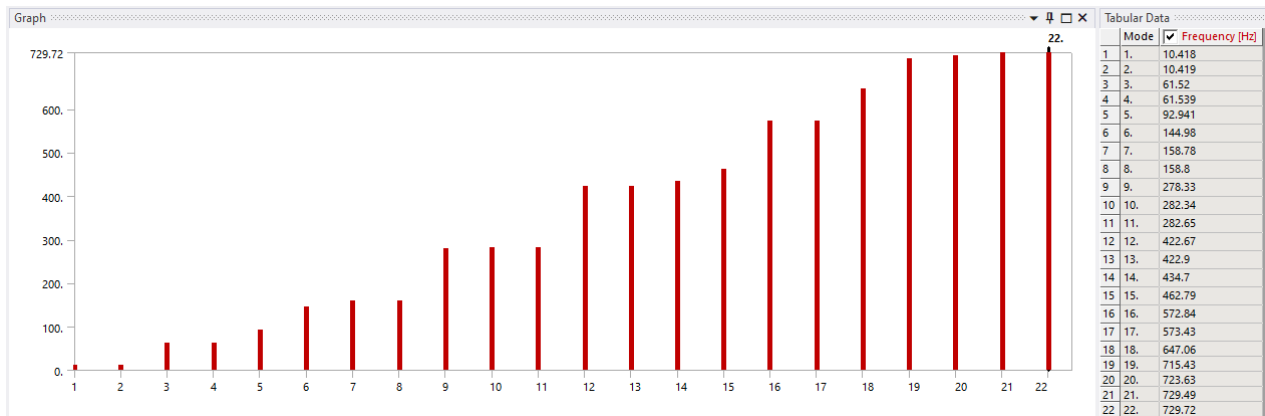


Figure 12. Modal analysis result.

Results (Part B)

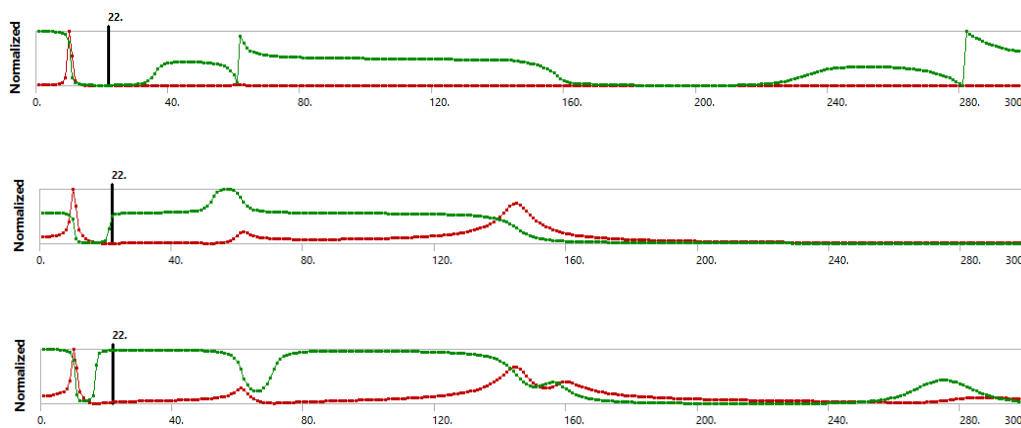
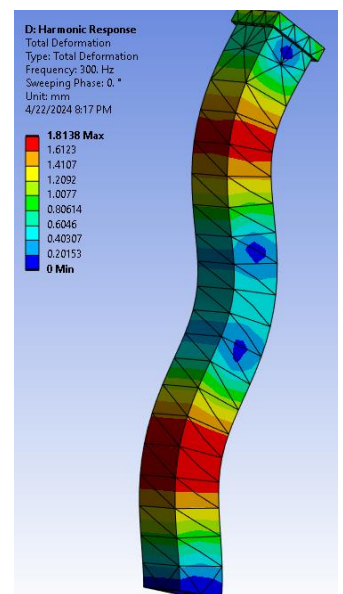


Figure 13. Deformation and normalized frequency response of X, Y, Z axis (top to bottom)



Results (Part C)

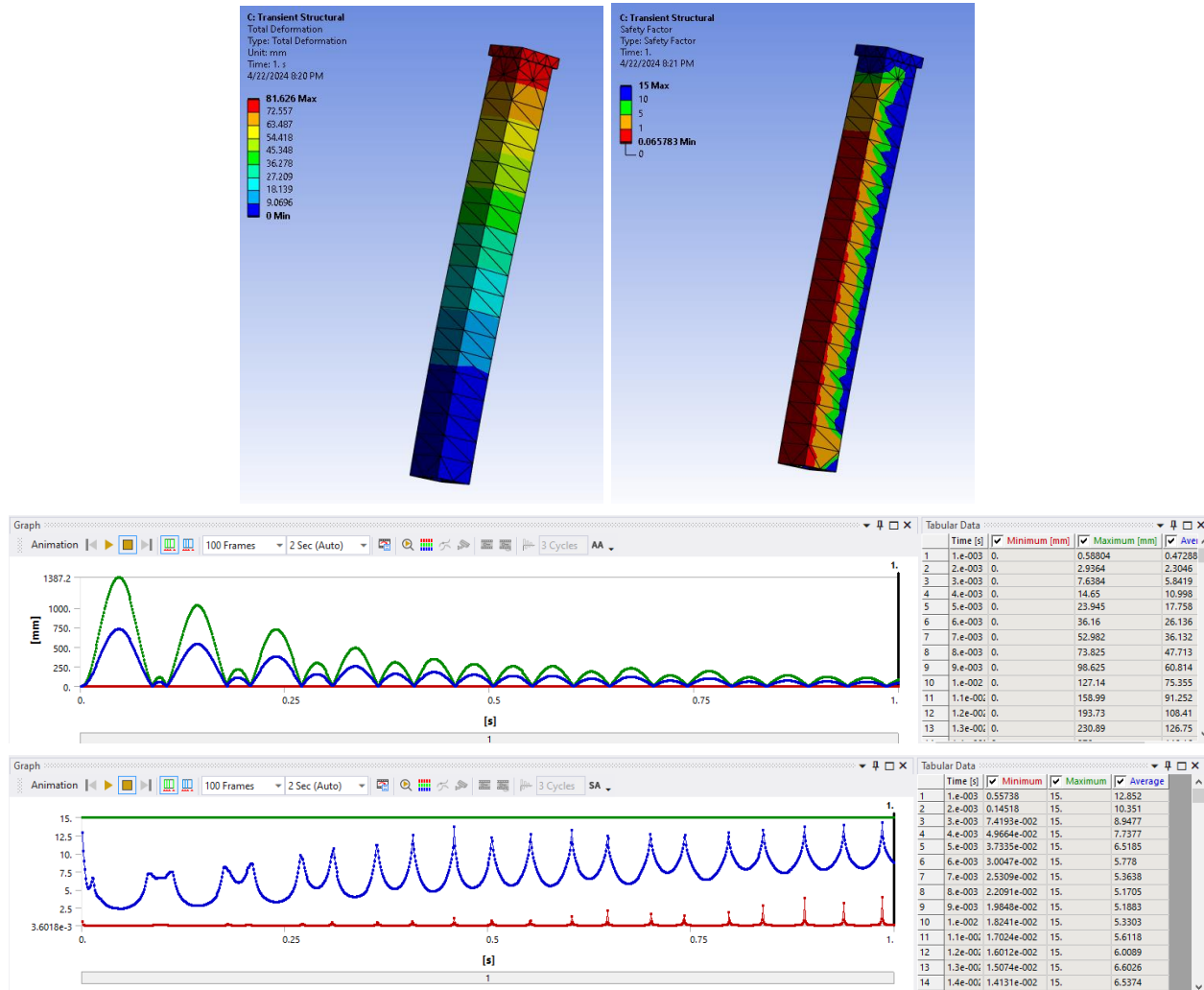


Figure 14. Deformation and safety factor under transient structural analysis

Conclusion

In this project, we conducted modal, harmonic, and transient analyses to comprehensively examine the dynamic behavior of the chimney structure.

These methodologies are crucial as they enable us to simulate and understand a wide array of real-world scenarios, particularly those characterized by transient conditions.

Furthermore, the convergence study, including selection of appropriate time steps and frequencies, along with trading-off computing time and accuracy, plays a significant role in the accuracy and reliability of these analyses.

This detailed approach allows for a more precise assessment of the chimney's response under various operational conditions, thereby ensuring its structural integrity and operational efficiency.