## Belt drive simulation

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### 1 Description of belt drive simulation

The examined belt drive has two pulleys  $P_1$  and  $P_2$  with identical radius and inertia, see the geometrical setup in Figure 1. The numerical modeling of the belt is based on the Absolute Nodal Coordinate Formulation (ANCF), [1]. The pulleys are simulated as rigid bodies while the contact between belt and pulleys is modeled as described in [2].

This numerical example is similar to the one developed in [3] with some modifications which attempt to eliminate the vibrations in the beginning of the simulation and allow the system to reach the steady state. The angular velocity of pulley  $P_1$  is prescribed by means of an algebraic constraint, while some resistance torque over time is added to pulley  $P_2$ , see the description hereafter. The

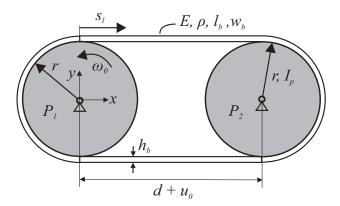


Figure 1: Belt drive with two pulleys, displaced from initial position by  $u_0$ .

Table 1: Main parameters for the belt drive

	Table 1: Main parameters for the best drive.			
Par.	Value	Units	Description	Names in code
$\overline{r}$	0.09995	m	pulley radius	radiusPulley
d	$0.1\pi$	m	distance between two pulleys	distancePulleys
$h_b$	0.0001	m	belt height	hc
$w_b$	0.08	m	belt width	ъ
$ar{l}_b$	$0.38\pi$	m	stress-free belt length	
$l_b$	$0.4\pi$	m	initial, deformed belt length	
$\varepsilon_{ref}$	-0.05	-	added reference axial strain	preStretch
EA	8000	${ m Nm}$	axial stiffness	EA
EI	$\frac{4}{3} \cdot 10^{-3}$	${ m Nm^2}$	bending stiffness	EI
$\rho$	1036	${ m kg/m^3}$	beam density	rhoA
dEA	1	$N/ms^2$	strain proportional damping	dEA
$\omega_{P1}$	12	$\rm rad~s^{-1}$	angular velocity of $P_1$	omegaFinal
$d_{P2}$	2	Nm/s	damping at $P_2$	rotationDampingWheels
$t_0$	0.05	s	driving start time	tAccStart
$t_1$	0.60	s	driving end time	tAccEnd
$t_{\tau 0}$	1.0	s	torque $\tau_{P2}$ starts	tTorqueStart
$t_{\tau 1}$	1.5	s	torque $\tau_{P2}$ reaches nominal value	tTorqueEnd
$I_p$	0.25	${ m kg}{ m m}^{-2}$	moment of inertia of pulleys	wheelInertia
<i>g</i>	9.81	${ m ms^{-2}}$	gravity	gVec

belt is modeled as Bernoulli-Euler beam with bending stiffness EI, axial stiffness EA, rectangular cross section with height  $h_b$  and width  $w_b$ , as well as stretch proportional damping, density, and further parameters given in Table 1. A constant acceleration is prescribed to pulley  $P_1$  between  $t_0$  and  $t_1$ :

$$\omega_{P1}(t) = \begin{cases}
0 \frac{\text{rad}}{\text{s}}, & \text{if } t < t_0 \\
\omega_{P1} \frac{t - t_0}{t_0 - t_1} & \text{if } t_0 < t < t_1 \\
\omega_{P1} & \text{else}.
\end{cases} \tag{1}$$

A torque proportional to the angular velocity is applied to the pulley  $P_2$  which represents

Table 2	· Dofault	values for	parameters
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Par.	Value	Units	Description	Name in code
$t_{end}$	2.45	s	evaluation time	P.tEnd
$\mu$	0.5	-	dry friction coefficient	P.dryFriction
$n_e$	240	-	number of elements	P.nANCFnodes
dt	$5 \cdot 10^{-5}$	S	time step size	P.stepSize
$n_{seg}$	4	-	number of segments	P.nSegments
$k_c$	$4 \cdot 10^{9}$	$N/m^3$	normal contact stiffness	contactStiffness
$\mu_k$	$5 \cdot 10^{9}$	$N/m^3$	tangential contact stiffness	frictionStiffness
$d_c$	$8 \cdot 10^{4}$	$Ns/m^3$	normal contact damping	contactDamping
$\mu_v$	$\sqrt{m_{seg}\mu_k} \approx$	$Ns/m^3$	tangential contact velocity penalty	frictionVelocityPenalty
	$\sqrt{m_{seg}\mu_k} \approx 3.22 \cdot 10^6$	·		

damping of rotational motion:

$$\tau_{P2}(t) = \begin{cases} 0 \text{ Nm}, & \text{if } t < 1\\ 25 (0.5 - 0.5 \cdot \cos(2(t - 1)\pi)) \text{ Nm} & \text{if } 1 < t < 1.5\\ 25 \text{ Nm} & \text{else} \,. \end{cases}$$
 (2)

As compared to [3], we use a much smaller belt height  $h_b$  in order to exclude bending effects, a higher pre-tension (due to pre-stretch), while keeping the axial stiffness EA the same. Furthermore, the bending stiffness is lowered by a factor of 50, which reduces bending effects, as it would lead to significant deviations from an analytical solution otherwise. The support of pulley  $P_1$  is not displaced during the first 0.05 s of the simulation, but the pre-stretch  $\varepsilon_{ref}$  is applied before running a static computation, which defines a static equilibrium for the dynamic simulation hereafter. The contact stiffness has been increased by a factor of 40 and a tangential stiffness (bristle) model has been included in order to retrieve highly accurate contact behavior.

## 2 Description of code

For simulating the system we are using the multibody dynamics code Exudyn [4]. The code in divided into sections (1, 2, ..., 8) and subsections (A, B, ...) for easier documenting and processing, see section 3 with subsections (A, B, ..., E):

- In section 1, we import necessary modules.
- Section 2 creates a multibody system, mbs.
- Section 3 consists of the Parameter Function. This function will be repeatedly called from Parameter Variation to update the value of the variables for which we perform variations.
  - We create a class P which contains all parameters for which we can perform Parameter variations, see Table 2. First the parameters are given their default values, see Table 2. Then we update the values of varying parameters through:

```
for key,value in parameterSet.items():
    setattr(P,key,value)
```

where setattr() is a Python function which sets the value of the attribute of an object.

- We create the model with respect to the parameter values given in Table 1.
- For the ANCF beam elements modeling the belt we are using ObjectANCFCable2D, see the documentation of Exudyn<sup>1</sup>, theDoc. Parameters used for ObjectANCFCable2D are defined in Table 3.
- For prescribing the angular velocity, we are using the following user function:

```
def UFvelocityDrive(mbs, t, itemNumber, 10ffset):
    if t < tAccStart: # driving start time
       v = 0
    if t >= tAccStart and t < tAccEnd:
       v = omegaFinal/(tAccEnd-tAccStart)*(t-tAccStart)
    elif t >= tAccEnd:
```

<sup>&</sup>lt;sup>1</sup>https://github.com/jgerstmayr/EXUDYN

Table 3: Input for ObjectANCFCable2D

1 3	
Input	Value
physicsMassPerLength	$\rho A$
${\tt physicsBendingStiffness}$	EI
${\tt physicsAxialStiffness}$	EA
physicsBendingDamping	dEI
${\tt physicsAxialDamping}$	dEA
${\tt physicsReferenceAxialStrain}$	$\varepsilon_{ref}$
physicsReferenceCurvature	0
${\tt useReducedOrderIntegration}$	2
${\tt strainIsRelativeToReference}$	False

v = omegaFinal
return v

- In section 4, simulation settings and visualization settings are defined.
- In section 5, we perform the static and dynamic equilibrium.
- In section 6, the obtained results are post-processed and saved in files.
- In section 7, one can choose between performing single simulation and performing parameter variation. An option for plotting figures is given. Solutions for parameters variations were stored in solution folder. Solutions from new runs are stored by default in solutionNosync.
- In section 8, saved results are plotted. Labels are generated. Cases given in iCases = [1, ..., 4] correspond to different varying quantities.

### 3 Installation and running

#### 3.1 Installing python and Exudyn

The code was tested in a Windows pc using Anaconda, 64bit, Python 3.7.6 and Spyder 4.0.1 which is included in the Anaconda installation.

For installing Exudyn PIP INSTALLER (pypi.org) was used based on the following instructions: Pre-built versions of Exudyn are hosted on pypi.org, see the project

• https://pypi.org/project/exudyn

As with most other packages, in the regular case (if your binary has been pre-built) you just need to  $do^2$ 

```
pip install exudyn
```

On Linux (currently only pre-built for UBUNTU, but should work on many other linux platforms), **update pip to at least 20.3** and use

```
pip3 install exudyn
```

For pre-releases (use with care!), add '--pre' flag:

```
pip install exudyn --pre
```

Results added <u>in src folder</u> were obtained using Exudyn V1.2.32.dev1. For installing this version do

```
pip install exudyn==1.2.32.dev1
```

For more information for installing Exudyn see the the $Doc^3$ .

#### 3.2 Running the code

Two python files are added in src folder. One for performing the belt drive simulation with the default values and another for performing variations and plotting figures. (The two files are identical with the only differences being in the flags which are enabling the operations of the code.)

For running these files the first option is to open an Anaconda prompt and copy paste the file location. The second option is to use Spyder and should be selected for making modifications in the code.

 $<sup>^2</sup>$ If the index of pypi is not updated, it may help to use pip install -i https://pypi.org/project/ exudyn

<sup>&</sup>lt;sup>3</sup>https://github.com/jgerstmayr/EXUDYN

### References

- [1] J. Gerstmayr and H. Irschik, "On the correct representation of bending and axial deformation in the absolute nodal coordinate formulation with an elastic line approach," *Journal of Sound and Vibration*, vol. 310, no. 3, pp. 461–487, 2008.
- [2] K. Ntarladima, M. Pieber, and J. Gerstmayr, "Contact modeling between axially moving beams and sheaves," under submission.
- [3] A. Pechstein and J. Gerstmayr, "A Lagrange-Eulerian formulation of an axially moving beam based on the absolute nodal coordinate formulation," *Multibody System Dynamics*, vol. 30, no. 3, pp. 343–358, 2013.
- [4] J. Gerstmayr, "Exudyn A C++ based Python package for flexible multibody systems," in the proceedings of the 6th Joint International Conference on Multibody System Dynamics and the 10th Asian Conference on Multibody System Dynamics, (New Delhi, India), 2022, submitted.