

Finite Element Method Implementation using PyGMSH meshing in Python

We aim to calculate the finite element solution of the partial differential equation given by:

$$-\left(\frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2}\right) = 1, \quad x^2 + y^2 < 1,$$

Subject to Dirichlet boundary conditions on the whole boundary, given by:

$$u(x, y) = 0, \quad x^2 + y^2 = 1$$

This boundary value problem has solution:

$$u(x, y) = \frac{1}{4}(1 - x^2 - y^2)$$

Bounded on the interval, $x^2 + y^2 = 1$.

We start by determining the weak solution of the PDE. As outlined by Whitley, Equation 7.10 defines the weak form of the PDE as follows:

$$\int_{\Omega} (\nabla v) \cdot (\nabla u) dA = \int_{\Omega} v dA$$

By Equation 7.25, the finite element solution is as follows:

$$U(x, y) = \sum_{j=1}^{N_{\text{node}}} U_j \phi_j(x, y)$$

By Equation 7.36, the local stiffness matrix is defined as follows:

$$A_{\text{local},ij}^{(k)} = \int_{e_k} (\nabla \phi_i) \cdot (\nabla \phi_j) dA = \int_{e_k} \left(\frac{\partial \phi_{k_i}}{\partial x} \frac{\partial \phi_{k_j}}{\partial x} + \frac{\partial \phi_{k_i}}{\partial y} \frac{\partial \phi_{k_j}}{\partial y} \right) dA = \int_{\Delta} \left(\frac{\partial \phi_{\text{local},i}}{\partial x} \frac{\partial \phi_{\text{local},j}}{\partial x} + \frac{\partial \phi_{\text{local},i}}{\partial y} \frac{\partial \phi_{\text{local},j}}{\partial y} \right) d\mathbf{e}$$

Where F , or the Jacobian/Transformation Matrix, is defined as follows:

$$F = \begin{pmatrix} \frac{\partial x}{\partial X} & \frac{\partial x}{\partial Y} \\ \frac{\partial y}{\partial X} & \frac{\partial y}{\partial Y} \end{pmatrix} = \begin{pmatrix} x_{k_2} - x_{k_1} & x_{k_3} - x_{k_1} \\ y_{k_2} - y_{k_1} & y_{k_3} - y_{k_1} \end{pmatrix}$$

Equation 7.41 simplifies the integral through the following:

$$A_{\text{local},ij}^{(k)} = \frac{1}{2} \det(F) \left(\frac{\partial \phi_{\text{local},i}}{\partial x} \frac{\partial \phi_{\text{local},j}}{\partial x} + \frac{\partial \phi_{\text{local},i}}{\partial y} \frac{\partial \phi_{\text{local},j}}{\partial y} \right)$$

The local load vector can be defined as follows:

$$b_{\text{local},i}^{(k)} = \int_{e_k} \phi_{k_i} dA = \int_{\Delta} \phi_{\text{local},i}(X, Y) \det(F) dA_x$$

Which can be updated as follows for our interval:

...

$$b_{local,i}^{(k)} = \int_0^{1-Y} \int_0^1 \phi_{local,i}(X,Y) \det(F) dx dy$$

Finally, the basis functions will be defined as follows for triangular elements:

$$\phi_1 = 1 - X - Y, \phi_2 = X, \text{ and } \phi_3 = Y$$

With the system defined, we now seek to define the mesh for the given interval. Using the Python library pygmsh, the 2-dimensional circle with radius 1 and center at (0,0) will be meshed. For reference, an example mesh output is included below

Mesh Size	N_x	N_y
1	13	13

Node ID	x	y	z
1	1	0	0
2	-0.5	0.866025	0
3	-0.5	-0.86603	0
4	0.766044	0.642788	0
5	0.173648	0.984808	0
6	-0.93969	0.34202	0
7	-0.93969	-0.34202	0
8	0.173648	-0.98481	0
9	0.766044	-0.64279	0
10	-0.26604	-0.22324	0
11	0.256771	0.215456	0
12	-0.38302	0.321394	0
13	0.25	-0.43301	0

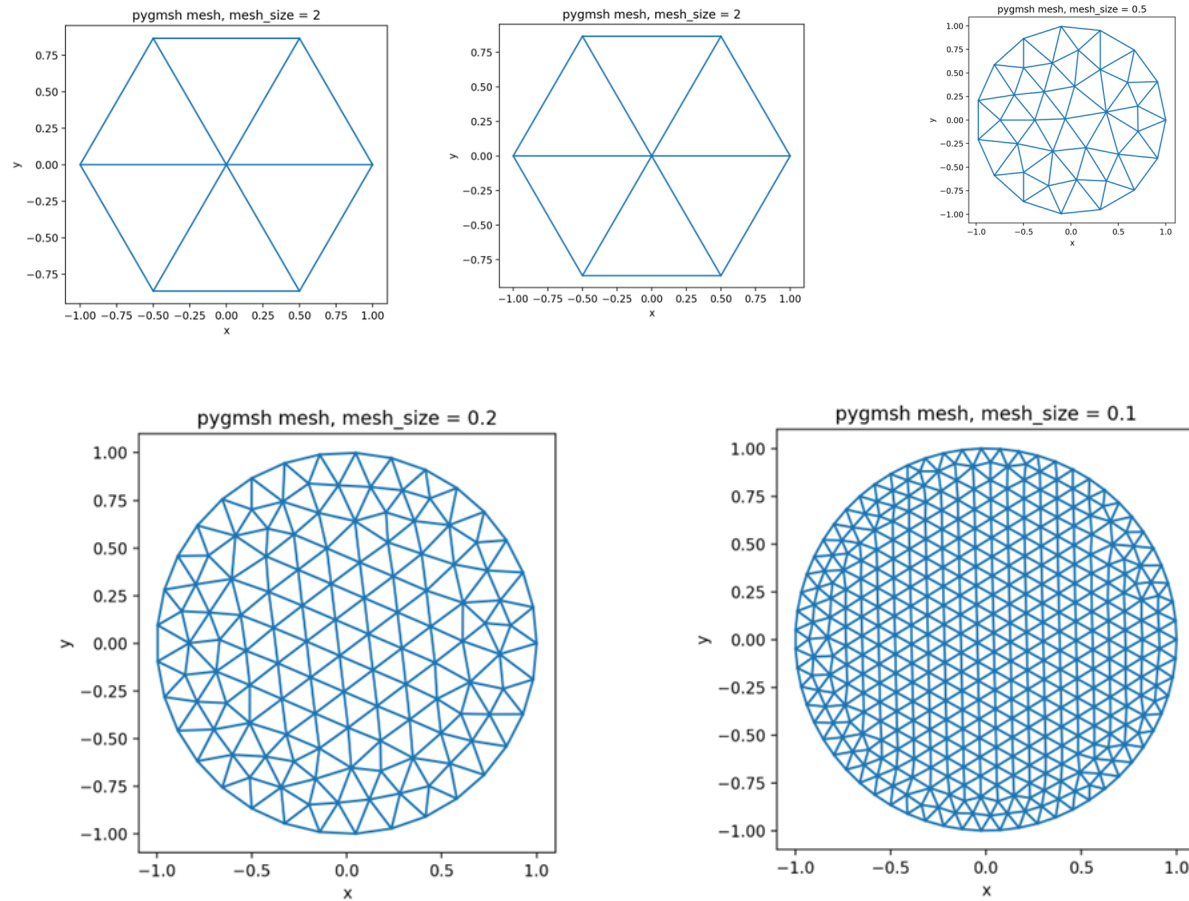
Element	Node 1	Node 2	Node 3
1	11	5	12
2	4	5	11
3	10	3	13
4	7	10	12
5	3	8	13
6	6	7	12
7	5	2	12
8	9	1	13
9	1	11	13
10	1	4	11
11	7	3	10
12	10	11	12
13	11	10	13
14	2	6	12
15	8	9	13

To interpret the meshsize refinement and how it relates to the number of nodes, the following table is presented.

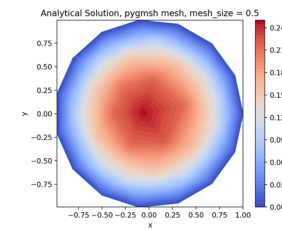
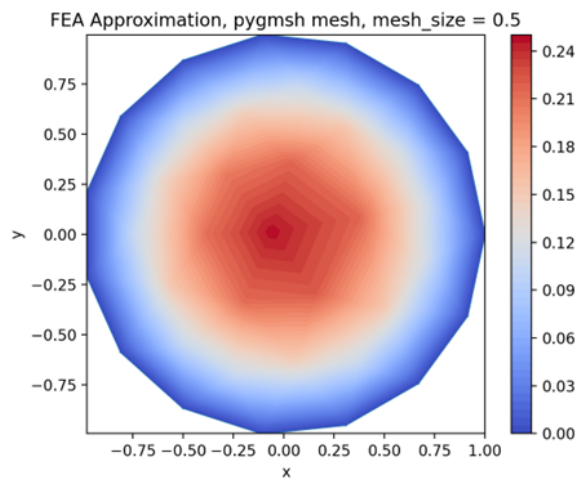
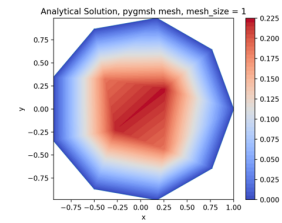
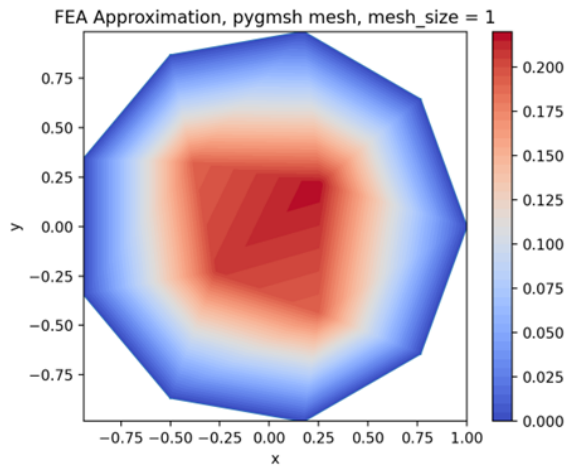
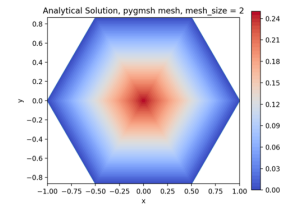
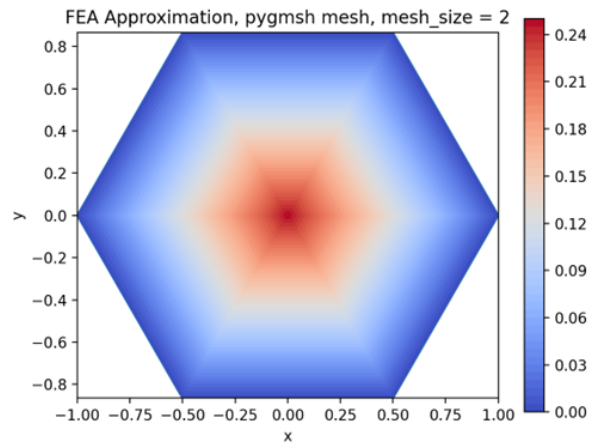
Mesh size	N_x	N_y
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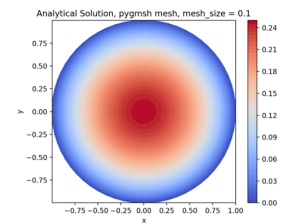
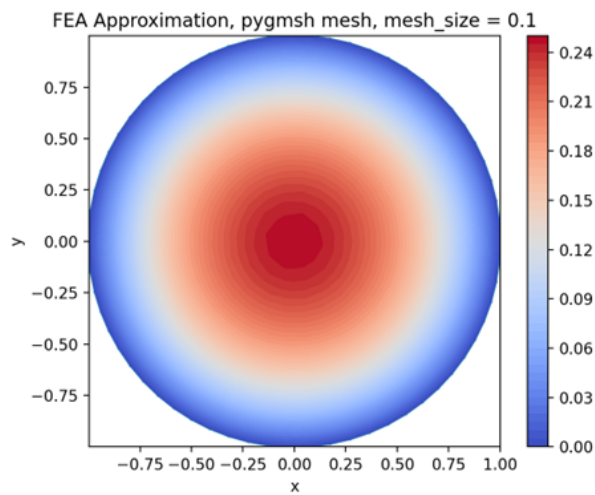
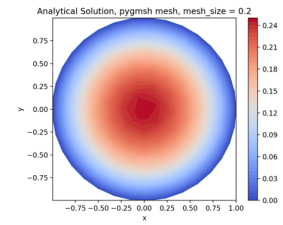
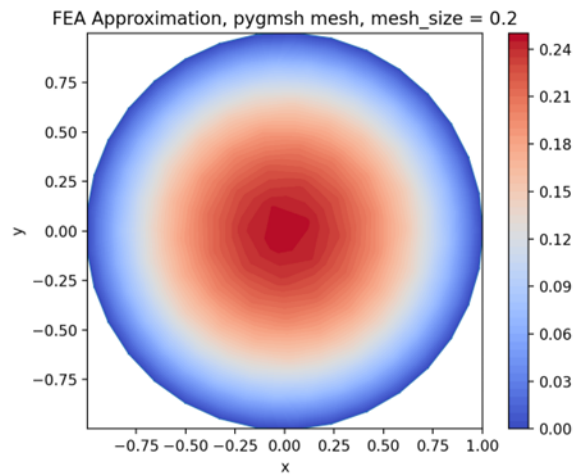
2	7	7
1	13	13
0.5	37	37
0.2	129	129
0.1	413	413

Now, varying mesh sizes are as follows:



Now, plotting the calculating solution using a contour plot for varying mesh sizes. The contour plots are included in pairs – the first plot is the FEA approximation and the second plot is the analytical solution.





Work Cited

Whiteley, J. (2017). *Finite element methods: A Practical Guide*. Springer.

Appendix

```
import pygmsh
import numpy as np
from sympy import symbols, integrate, simplify
import matplotlib.pyplot as plt
from sympy import Matrix
import matplotlib.tri as tri
import time

X=symbols('X')
Y=symbols('Y')
```

```

def u_analytical(x,y):
    return (0.25)*(1-(x**2)-(y**2))

size = 2
with pygmsh.geo.Geometry() as geom:
    r = 1.0
    circle = geom.add_circle([0, 0, 0], r, mesh_size=size)
    mesh = geom.generate_mesh()

point = mesh.points
x= mesh.points[1:,0]
y= mesh.points[1:,1]
loca = mesh.cells_dict["triangle"]

#Determining Nx and Ny
Nx = len(np.unique(x))
print(f"Nx: {Nx}")
Ny = len(np.unique(y))
print(f"Ny: {Ny}")

#Defining size of global matrix and vector
n=len(point[:,0])-1

#Define global force vector
K=np.zeros((n,n),dtype=float)

#Define global stiffness matrix
f=np.zeros((n,1),dtype=float)

#Building locals and adding to globals for each element
for e in range(len(loca[:,0])):

    #Extracting physical dimensions for each node
    nodes_element=loca[e]

    #Basis functions for triangle mesh
    phi = Matrix([1 - X - Y, X, Y])

    #Defining local stiffness matrix at each element
    K_local=np.zeros((3,3),dtype=float)

    f_local=np.zeros((3,1),dtype=float)

    #Extracting physical dimensions from mesh
    coords=np.array([mesh.points[nodes_element[0]],mesh.points[nodes_element[1]],mesh.points[nodes_element[2]]])
    xk1,yk1=coords[0,:2]
    xk2,yk2=coords[1,:2]
    xk3,yk3=coords[2,:2]

    #Jacobian from physical dimensions
    Jac = np.array([[xk2 - xk1, xk3 - xk1],
                    [yk2 - yk1, yk3 - yk1]])
    detJ = np.linalg.det(Jac)
    invJT = np.linalg.inv(Jac).T # J^{-T}

```

```

#Defining gradient
gradient = np.array([[1,-1],[1,0],[0,1]])

#Defining gradient in physical dimensions
gradient_X_Y = gradient@invJT.T

#Determining area based on determinat
area = abs(detJ)/2

#Defining Local for each element
for i in range(0,3):
    for j in range(0,3):
        K_local[i,j] = np.dot(gradient_X_Y[i], gradient_X_Y[j])*area
    aux2= simplify(phi[i]*float(abs(detJ)))
    f_local[i]=float(integrate(integrate(simplify(aux2), (X, 0, 1 - Y)), (Y, 0, 1)))

#Adding locals to globals
for i in range(3):
    for j in range(3):
        K[nodes_element[i]-1,nodes_element[j]-1]+=K_local[i,j]
    f[nodes_element[i]-1]+=f_local[i]

#Boundary Conditions
zeros = []
aux=1e-6
i = 0
#Determining where boundary conditions should apply
for (x,y,_) in (mesh.points):
    if abs(x**2+y**2-1)<aux:
        zeros.append(i)
    i+=1

#Applying the boundary conditions
for i in zeros:
    K[i-1,:]=0
    K[:,i-1]=0
    f[i-1]=0
    K[i-1,i-1]=1

#Backslash
ts_bs_1 = time.time()
u=np.linalg.solve(K,f)
ts_bs_2 = time.time()
print(f"Backslash Performance:{ts_bs_2-ts_bs_1}")

#Regular matrix multiplication
ts_l_1=time.time()
u = np.linalg.inv(K)@f
ts_l_2 = time.time()
print(f"Linear Algebra Performance:{ts_l_2-ts_l_1}")

u =np.vstack([np.zeros((1,1)),u])

```

```

#Determining analytical solution
x_analytical =mesh.points[:,0]
y_analytical =mesh.points[:,1]
u_an=u_analytical(x_analytical,y_analytical)

#Processing u for plotting
u_flat=u.flatten()
x= mesh.points[:,0]
y= mesh.points[:,1]
triangle=tri.Triangulation(x,y,loca)

#Plotting FEA Approximation
plt.triplot(x, y, loca)
plt.gca().set_aspect("equal")
cont = plt.tricontourf(triangle,u_flat, levels=50, cmap="coolwarm")
plt.colorbar(cont)
plt.title(f"FEA Approximation, pygmsh mesh, mesh_size = {size}")
plt.xlabel('x')
plt.ylabel('y')

#Plotting Analytical Solution
plt.figure(2)
triangle=tri.Triangulation(x,y,loca)

plt.triplot(x, y, loca)
plt.gca().set_aspect("equal")
cont2 = plt.tricontourf(triangle,u_an, levels=50, cmap="coolwarm")
plt.colorbar(cont2)
plt.title(f"Analytical Solution, pygmsh mesh, mesh_size = {size}")
plt.xlabel('x')
plt.ylabel('y')

plt.show()

```