GLASGOW COLLEGE UESTC

Exam

Dynamics and Control (UESTC3001)

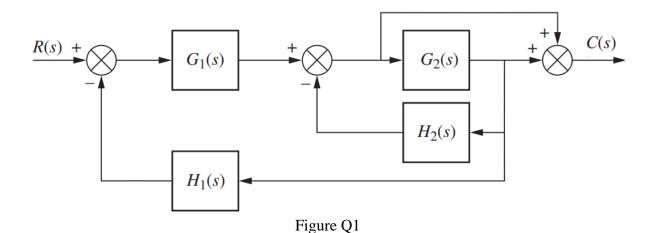
Date: 27th June 2021 Time: 09:30-11:30AM

Time: 2h

Attempt All Questions. Total 100 marks

- All questions bear equal marks [25 marks]
- Use one answer sheet for each of the questions in this exam.
- Show all work on the answer sheet.
- Make sure that your University of Glasgow and UESTC Student Identification Numbers are on all answer sheets.
- An electronic calculator may be used provided that it does not allow text storage or display, or graphical display.
- All graphs should be clearly labelled and sufficiently large so that all elements are easy to read.
- The numbers in square brackets in the right-hand margin indicate the marks allotted to the part of the question against which the mark is shown. These marks are for guidance only.

- a) Explain the primary reasons for using control systems in engineering applications. [4]
- b) The block diagram of a control system is shown in Figure Q1. Using block diagram reduction techniques, determine the closed-loop transfer function of the output C(s) over the command R(s).



i) Redraw the simplified block diagram.

- [5 marks]
- ii) Reduce the block diagram to a single block using block reduction techniques.

 [8 marks]
- c) Given the transfer function below, use the Routh's criteria to determine the range of values of *K* for which the system remains stable.

Transfer function =
$$\frac{s^3 + 3s^2 + s + 6}{s^4 + 12s^3 + s^2 + 4s + K}$$
 [8 marks]

Q2.

In order to regulate the speed of a rotor ω , a control system has been designed. The equation linking the torque T(t) and the angular speed $\omega(t)$ is given below:

$$T(t) = 0.1\dot{\omega}(t) + 5\omega(t)$$

The controller has a transfer function $G_c(s)$ which operates on the error between the measured voltage $v_m(t)$ representing the actual speed of the rotor $\omega(t)$ and the voltage $v_r(t)$ representing the required speed $\omega_r(t)$. The voltage $v_m(t)$ supplied by the sensor is proportional to the actual speed of the rotor $\omega(t)$.

$$v_m(t) = \frac{1}{2\alpha}\omega(t)$$

The cable linking the output of the sensor to the comparator is not shielded, thus a noise N(t) is added to the measured voltage $v_m(t)$. The required voltage $v_r(t)$ is computed mathematically from the required speed $\omega_r(t)$ using the same relationship as the sensor. The controller (via an electric servo motor) supplies the driving torque T(t).

Consider the steps described in parts (a) to (d) below, then draw the block diagram in (e):

- (a) Identify all the variables as a function of the complex variable s that are a part of the system. [2]
- (b) Translate the transfer function for torque and output angular speed from time domain to the s-domain. [2]
- (c) Translate the transfer function for the measured speed and the output angular speed from the time domain to the s-domain. [2]
- (d) Draw the relations linking the variables to each other based on the description in the text. [6]
- (e) Draw the block diagram of the system described above. [7]
- (f) A proportional + integral + derivative controller is used with constant proportional and integral gain, K_p and K_i . What is the effect of increasing the derivative gain on the system response (natural frequency, frequency, oscillation and damping) to a unit step input for a first-order system? [6]

Q3. Consider the control system shown in Figure Q3.

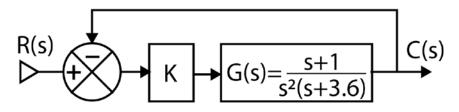


Figure Q3

- (a) Using the plant transfer function $G(s) = \frac{s+1}{s^2(s+3.6)}$ in a closed-loop system with unit negative feedback with a controller gain of K, explain what is meant by root-locus analysis of a system KG(s)? Explain the relevance of the Characteristic Equation of the system when considering such analysis. [6]
- (b) Before sketching the root-locus diagram for the open loop feedback system with the plant transfer function G(s) given above in Figure Q3, considering a unit negative feedback loop with a controller gain of K > 0.

Show all calculations, including the following:

- i) The values of the poles and zeros and the order of the open loop system [3]
- ii) The intersection point and angle of the asymptotes [4]
- iii) The breakaway point [5]
- iv) Investigate if there are any points where root locus branches cross the imaginary axis. [3]
- v) Sketch of the root loci for this system. [4]

Q4.

a) Consider the open-loop transfer function of a process given by:

$$G(s) = \frac{K}{s(s-5)(s+100)}$$

Answer the following questions for K = 300 (you are expected to fully justify your answers).

(i) Sketch the Nyquist plot for G. As part of this question show clearly the calculations for both the gain and phase of the system at the key frequencies. [10]

(<u>Hint to draw the Nyquist plot</u>: You may need to consider the real part of the transfer function and its limit when ω tends to 0 to trace the sketch accurately.)

(ii) Calculate the gain and phase margins. You must show your calculations and reasoning. [8]

Hint: The following factorisation may be useful

$$\omega^{6} + 10025\omega^{4} + 2.5 \cdot 10^{5}\omega^{2} - 9 \cdot 10^{4}$$

= $(\omega + 0.5958)(\omega - 0.5958)(\omega^{2} + 10^{4})(\omega^{2} + 25.356)$

- (iii) Based on your work in (i) and (ii), conclude on the stability of the closed loop system. [2]
- b) A closed-loop system has the following characteristic equation:

$$s^4 + 8s^3 + 8s^2 + (K+1)s + Ka = 0$$

where K is a gain term.

Using Routh-Hurwitz, find the conditions that K and a must meet for the closed-loop to be stable. [5]

Data Sheet: Important Laplace Transform Pairs

Function f(t)	Laplace Transform F(s)
Unit step function $u(t)$	$\frac{1}{s}$
t^n	$\frac{n!}{s^{n+1}}$
e^{-at}	$\frac{1}{s+a}$
te ^{-at}	$\frac{1}{(s+a)^2}$
$\frac{1}{a} \Big[1 - e^{-at} \Big]$	$\frac{1}{s(s+a)}$
$\sin \omega t$	$\frac{\omega}{s^2 + \omega^2}$
cos ωt	$\frac{s}{s^2 + \omega^2}$
$e^{-at}\sin \omega t$	$\frac{\omega}{(s+a)^2+\omega^2}$
$e^{-at}\cos\omega t$	$\frac{s+a}{(s+a)^2+\omega^2}$
Unit impulse function $\delta(t)$	1

Data sheet 2: Rules for block diagram reduction

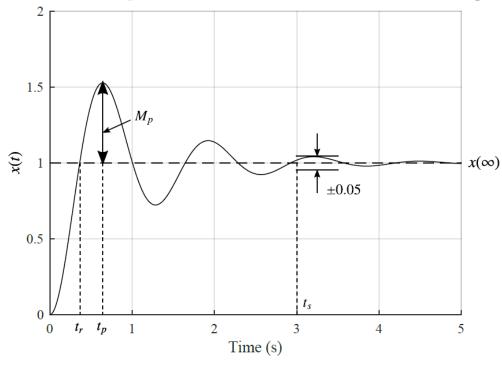
Original Rule Equivalent 1. Cascaded blocks $G_1(s)G_2(s)$ 2. Summing two $X_3(s)$ $X_1(s)$ $X_2(s)$ $X_3(s)$ $G(s) \pm 1$ signals $X_1(s)$ $X_3(s)$ $X_1(s)$ $X_3(s)$ G(s)3. Moving a G(s)summing point $X_2(s)$ behind a block $X_2(s)$ G(s) $X_1(s)$ $X_1(s)$ $X_3(s)$ 4. Moving a G(s)summing point ahead of a block $X_2(s)$ $\overline{G(s)}$ $X_1(s)$ $X_1(s)$ 5. Moving a branch G(s)G(s)point ahead of a G(s)block $X_2(s)$ $X_2(s)$ $X_2(s)$ $X_1(s)$ $X_2(s)$ G(s)G(s)6. Moving a branch point behind a block $X_1(s)$ $X_1(s)$ $X_2(s)$ G(s) $X_1(s)$ $X_2(s)$ 7. Eliminating a G(s)feedback loop $1\mp G(s)H(s)$ B(s)H(s)

Datasheet 3: Second order systems

The general form of the second-order system is

$$\ddot{x}(t) + 2\zeta\omega_n\dot{x}(t) + \omega_n^2x(t) = Ku(t), \qquad \qquad \frac{X(s)}{U(s)} = \frac{K}{s^2 + 2\zeta\omega_n s + \omega_n^2}$$

where ζ is the **damping ratio** and ω_n is the **natural frequency**, as stated previously.



Characteristics of an underdamped second-order system in response to a step input.

Characteristics of the Response of an Underdamped 2nd Order System to a Step Input

Rise Time
$$t_r = \frac{1}{\omega_d} \tan^{-1} \left(\frac{\sqrt{1 - \zeta^2}}{\zeta} \right)$$

$$Peak Time \qquad t_p = \frac{\pi}{\omega_d}$$

Maximum Overshoot
$$M_p = x(\infty)e^{-\left(\pi \xi/\sqrt{1-\zeta^2}\right)}$$
 or $M_p = 100e^{-\zeta\omega_n t_p}$ %

Settling Time
$$t_s = \frac{3}{\zeta \omega_n}$$

Damped natural Frequency
$$\omega_d = \omega_n \sqrt{1 - \zeta^2}$$

Datasheet 4: Root Locus

Root Locus Analysis

Properties of the Open Loop Transfer Function

The characteristic equation of a system may be written in the form: 1 + F(s) = 0 where F(s) is the open loop transfer function. This gives the magnitude condition that

$$|F(s)| = 1$$

and given that he open loop transfer function takes the form:

$$F(s) = \frac{K(s - z_1)(s - z_2) \cdots (s - z_v)}{s^n(s - p_1)(s - p_2) \cdots (s - p_u)}$$

we can write

$$K = \frac{\prod_{j=1}^{n+u} |s - p_j|}{\prod_{j=1}^{v} |s - z_j|}$$

Sketching a Root Locus

Angle of Asymptotes: $\phi = \frac{(2m+1)\times 180}{P-Z}$ degrees for m = 0, 1, 2, ... (P-Z-1)

The intersection point: $\sigma_A = \frac{\sum_{j=1}^{P} \text{Re}(p_j) - \sum_{i=1}^{Z} \text{Re}(z_i)}{P - Z}$

Breakaway Point: $\frac{dK}{ds} = 0$

Departure Angle from a Complex Pole, ϕ_d , is given by:

 ϕ_d =180°- Σ angles of vectors to the complex pole from all other poles + Σ angles of vectors to the complex pole from all zeros.

Arrival Angle to a Complex Zero, ϕ_a , is given by:

 φ_a =180- Σ angles of vectors to the complex zero from all other zeros + Σ angles of vectors to the complex zero from all poles

Newton-Raphson Method: For a function A(s) = 0, successive estimates of a root may be obtained from:

$$S_{i+1} = S_i - \frac{A(S_i)}{A'(S_i)}$$

Datasheet 5: Frequency Response Analysis

General theory

For a system with closed loop transfer function: $G(s) = \frac{X(s)}{Y(s)}$

a periodic input

$$y(t) = Y \sin \omega t$$

yields a steady state response $x(t)_{ss} = Y |G(i\omega)| \sin(\omega t + \phi)$

where

$$|G(i\omega)|$$
 = the gain of the system,

and

$$\phi = \angle G(i\omega)$$
 = the phase of the system.

Elements of the Bode plot

Factor	G(s)	G(i\omega)	$ G(i\omega) _{dB}$	$\angle G(i\omega)$
			dB	rad
Gain	K	K	20 log ₁₀ K	0
Poles at Origin	$\frac{1}{s^n}$	$\frac{1}{i^n \omega^n}$	$-20n\log_{10}\omega$	$-n\frac{\pi}{2}$
Zeros at Origin	S^{n}	$i^{\eta_1} \mathcal{O}^{\eta_1}$	$20n\log_{10}\omega$	$n\frac{\pi}{2}$
Pole	1	1	<i>ω</i> →0: 0	<i>ω</i> →0: 0
	$\frac{1}{1+as}$	$1+i\omega\tau$	$\omega \to \infty: \\ -20 \log_{10}(\tau \omega)$	$\omega \to \infty : -\frac{\pi}{2}$ $\omega \to 0 : 0$
Zero	$1 + \tau s$	$1 + i\omega \tau$	$\omega \rightarrow 0: 0$	$\omega \rightarrow 0: 0$
			$\omega \to \infty:$ $20 \log_{10}(\tau \omega)$	$\omega \to \infty$: $\frac{\pi}{2}$
01	2	2	$\omega \rightarrow 0: 0$	$\omega \rightarrow 0: 0$
Quadratic Poles	$\frac{\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2}$	$\frac{\omega_n^2}{\left(\omega_n^2 - \omega^2\right) + 2\zeta\omega_n\omega i}$	$\omega \to \infty: \\ -40 \log_{10}(\tau \omega)$	<i>ω</i> →∞: -π
Overdueti -	2 2 4 2	(2 2) 2%	<i>ω</i> →0: 0	<i>ω</i> →0: 0
Quadratic Zeros	$s^2 + 2\zeta\omega_n s + \omega_n^2$	$\left(\omega_n^2 - \omega^2\right) + 2\zeta\omega_n\omega i$	$\omega \to \infty :$ $40 \log_{10}(\tau \omega)$	ω→∞: π

Properties of the Bode Plot

For quadratic poles: Resonant Frequency,
$$\omega_r$$
: $\omega_r =$

Resonant Frequency,
$$\omega_r$$
: $\omega_r = \omega_n \sqrt{1 - 2\zeta^2}$

$$M_r = \frac{1}{2\zeta\sqrt{1 - \zeta^2}}$$