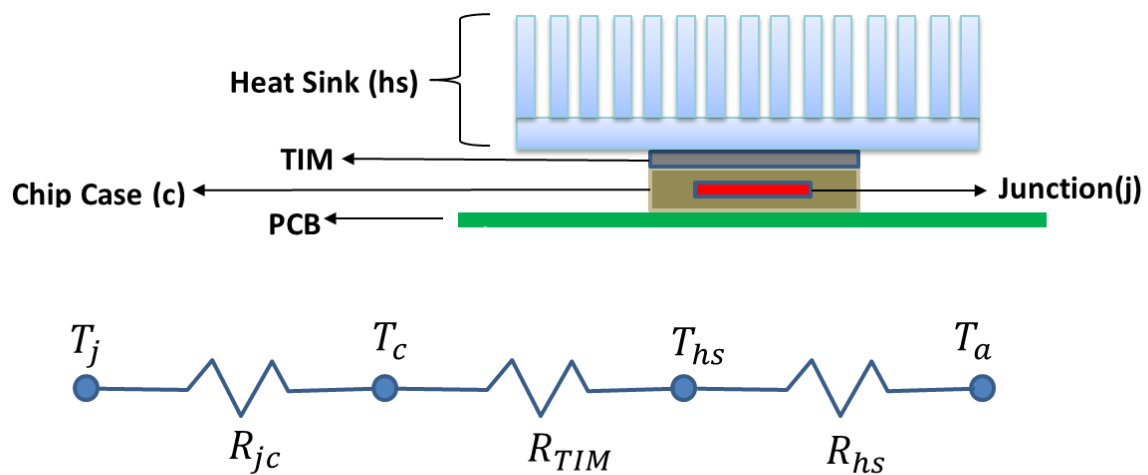


# REPORT



$$R_{total} = R_{jc} + R_{TIM} + R_{hs}$$

Where,

$$R_{hs} = R_{cond} + R_{conv}$$

Schematic diagram for the Heat Sink

## Step 1: Given Data and Assumptions

**Processor Die Dimensions:**

processor specification				
Die Length	52.5	mm	0.0525	m
Die width	45	mm	0.045	m
Die thickness	2.2	mm	0.0022	m
Die Area	2362.5	mm <sup>2</sup>	0.002363	m <sup>2</sup>

**Thermal Design Power (TDP):** Q=150 W

**Heat Sink Specifications:**

Sink Length	90	mm	0.09	m
sink width	116	mm	0.116	m
base thickness	2.5	mm	0.0025	m
base area	10440	mm	0.01044	m <sup>2</sup>
No. of Fins	60			
Fin Thickness	0.8	mm	0.0008	m
overall height	27	mm	0.027	m
Fin Height	24.5	mm	0.0245	m
Fin Spacing	1.152542	mm	0.001153	m

**Assumptions:**

- Heat sink material: **Aluminum (Al 6061-T6)** With Thermal conductivity,  $k_{Al} = 167 \text{ W/m} \cdot \text{K}$

- Thermal interface material:
- **Thermal grease with thermal conductivity  $k_{TIM}=4 \text{ W/m} \cdot \text{K}$**
- Assuming a **thermal grease layer of 0.1 mm thickness**
- Cooling Medium: Air at 25°C
- At 25°C, properties of Air are

Thermal Conductivity	0.0262	$\text{W/mk}$
Kinematic viscosity	$1.57E-05$	$\text{m}^2/\text{s}$
Prandtl Number	0.71	
Velocity of Air	1	$\text{m/s}$

## Step 2: Define the Thermal Resistance Network

The **total thermal resistance**  $R_{total}$  from the processor junction to ambient consists of:

$$R_{total} = R_{jc} + R_{TIM} + R_{hs}$$

where:

$R_{jc}$  is the Junction to case resistance within the processor.

$R_{TIM}$  is the resistance due to thermal interface material (TIM)

$R_{hs}$  is the thermal resistance due to heat sink, which consist of  $R_{conduction}$ , conduction resistance through heat sink base and  $R_{convection}$  accounts for heat loss through forced convection.

## Step 3: Junction-to-Case Resistance calculation ( $R_{jc}$ )

For modern processors, **junction-to-case resistance** is often provided in datasheets.

A typical value:  $R_{jc} = 0.1 \text{ } ^\circ\text{C/W}$

## Step 4: Case-to-TIM Resistance

The resistance of the thermal interface material (TIM) is given by:

$$R_{TIM} = \frac{t_{TIM}}{k_{TIM}A_{die}}$$

Where

$t_{TIM}$  = Thickness of TIM

$A_{die}$  = Area of die

## Step 5: Thermal resistance due to heat sink $R_{hs}$

### Conduction Resistance through Heat Sink Base ( $R_{conduction}$ )

The conduction resistance through the **aluminum base** is:

$$R_{cond} = \frac{t_b}{k_{Al}A_{die}}$$

Where  $t_b$  is thickness of base.

And  $A_{die}$  = Area of die

### Convection Resistance ( $R_{convection}$ )

Reynolds number is given by:

$$Re = \frac{VL}{\nu}$$

Where

$V$  = Air velocity

$L$  = Characteristic length, here Fin spacing  $S_f$

$$S_f = \text{Heat sink Width} - \frac{(N_{fins} \times \text{Fin Thickness})}{N_{fin} - 1}$$

$\nu$  = Kinematic viscosity

The heat sink fins create channel flow, similar to flow between parallel plates.

For low  $Re$  ( $< 2300$ ), flow remains laminar and developing, **so** Sieder-Tate is applicable.

$$Nu = 1.86 \left( Re \times Pr \times \frac{2S_f}{L} \right)^{\frac{1}{3}}$$

For higher  $Re$  ( $> 2300$ ), the flow transitions to turbulent, where the Dittus-Boelter equation is more appropriate so Nusselt Number is:

$$Nu = 0.023 Re^{0.8} Pr^{0.3}$$

So

Convective Heat transfer coefficient

$$h_{convection} = \frac{Nu \times k_{air}}{2S_f} \quad W/m^2K$$

Now,

$$R_{convection} = \frac{1}{h_{convection} \times A_{total}}$$

Where

$A_{total}$  = Total area for convection

And Thermal resistance through Heat sink,

$$R_{hs} = R_{conduction} + R_{convection}$$

## Step 6: Total Thermal Resistance

$$R_{total} = R_{jc} + R_{TIM} + R_{hs}$$

## Step 7: Calculation Junction Temperature

$$T_{junction} = T_{ambient} + QR_{total}$$

## Result from excel: (Reference)

Excel Results		
Total Heat Sink Resitance	0.373043	degC/W
Junction Temperature	80.95652	degC