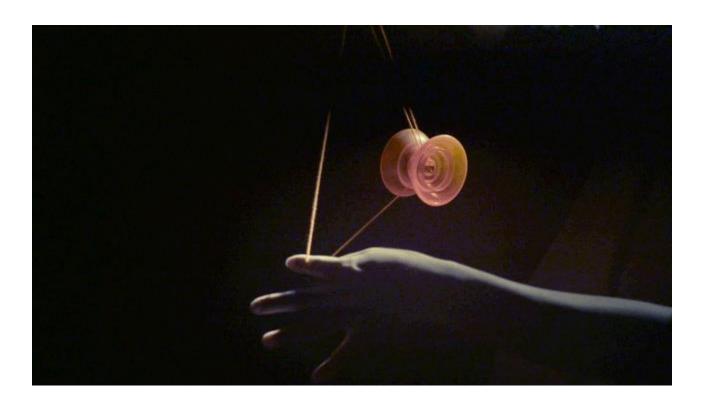
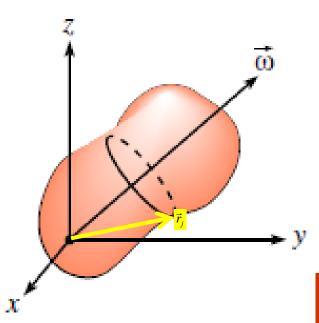
RIGID BODY IN MOTION



Angular momentum of a rigid body



$$[L] = [I][\omega]$$

$$\begin{pmatrix} L_x \\ L_y \\ L_z \end{pmatrix} = \begin{pmatrix} I_{xx} & I_{xy} & I_{xz} \\ I_{yx} & I_{yy} & I_{yz} \\ I_{zx} & I_{zy} & I_{zz} \end{pmatrix} \begin{pmatrix} \omega_x \\ \omega_y \\ \omega_z \end{pmatrix}$$

$$[L] = [I][\omega]$$
 Equivalent to $\overrightarrow{p} = \overrightarrow{mv}$

$$\vec{p} = \vec{mv}$$



$$L_{x} = I_{xx}\omega_{x} + I_{xy}\omega_{y} + I_{xz}\omega_{z}$$

$$L_{y} = I_{yx}\omega_{x} + I_{yy}\omega_{y} + I_{yz}\omega_{z}$$

$$L_z = I_{zx}\omega_x + I_{zy}\omega_y + I_{zz}\omega_z$$

The fact that I is a matrix means that L and ω do not necessarily point in the same direction.

$$\begin{bmatrix} I \end{bmatrix} = \begin{pmatrix} I_{xx} & I_{xy} & I_{xz} \\ I_{yx} & I_{yy} & I_{yz} \\ I_{zx} & I_{zy} & I_{zz} \end{pmatrix}$$

$$I_{xx} = \sum m_j \left(y_j^2 + z_j^2 \right)$$

For a continuous medium,

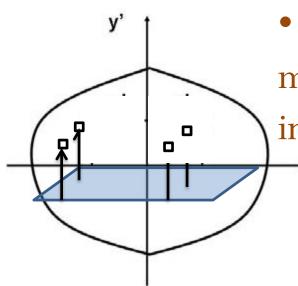
$$I_{xx} = \int (y^2 + z^2) dm$$

$$[I] = \begin{pmatrix} \int (y^2 + z^2)dm & -\int xydm & -\int zxdm \\ -\int xydm & \int (z^2 + x^2)dm & -\int yzdm \\ -\int zxdm & -\int yzdm & \int (x^2 + y^2)dm \end{pmatrix}$$

$$\begin{bmatrix} I \end{bmatrix} = \begin{pmatrix} \int (y^2 + z^2)dm & -\int xydm & -\int zxdm \\ -\int xydm & \int (z^2 + x^2)dm & -\int yzdm \\ -\int zxdm & -\int yzdm & \int (x^2 + y^2)dm \end{pmatrix}$$

- We observe that the quantity in the integrand of diagonal terms is precisely the square of the distance to the x, y and z axis, respectively. They are analogous to the moment of inertia used in two-dimensional case.
- Off diagonal elements in the inertia matrix are a measure of the **imbalance** in the mass distribution.

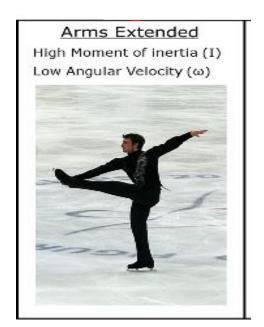
$$[I] = \begin{pmatrix} \int (y^2 + z^2)dm & -\int xydm & -\int zxdm \\ -\int xydm & \int (z^2 + x^2)dm & -\int yzdm \\ -\int zxdm & -\int yzdm & \int (x^2 + y^2)dm \end{pmatrix}$$



• Off diagonal elements in the inertia matrix are a measure of the **imbalance** in the mass distribution.

 \mathbf{x} For symmetric object, $\mathbf{I}_{\mathbf{x}\mathbf{y}}$ will be zero

$$\begin{bmatrix} I \end{bmatrix} = \begin{pmatrix} \int (y^2 + z^2)dm & -\int xydm & -\int zxdm \\ -\int xydm & \int (z^2 + x^2)dm & -\int yzdm \\ -\int zxdm & -\int yzdm & \int (x^2 + y^2)dm \end{pmatrix}$$



Product of inertia will be non-zero.

Angular momentum and Torque

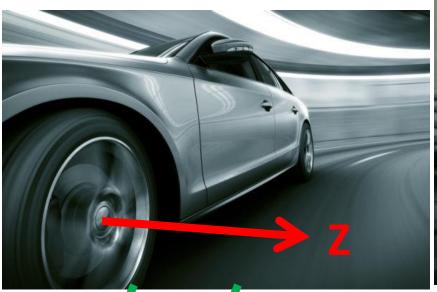
$$\begin{pmatrix} L_{x} \\ L_{y} \\ L_{z} \end{pmatrix} = \begin{pmatrix} I_{xx} & I_{xy} & I_{xz} \\ I_{yx} & I_{yy} & I_{yz} \\ I_{zx} & I_{zy} & I_{zz} \end{pmatrix} \begin{pmatrix} \omega_{x} \\ \omega_{y} \\ \omega_{z} \end{pmatrix} = \begin{pmatrix} \tau_{x} \\ \tau_{y} \\ \tau_{z} \end{pmatrix} = \begin{pmatrix} I_{xx} & I_{xy} & I_{xz} \\ I_{yx} & I_{yy} & I_{yz} \\ I_{zx} & I_{zy} & I_{zz} \end{pmatrix} \begin{pmatrix} \alpha_{x} \\ \alpha_{y} \\ \alpha_{z} \end{pmatrix}$$

$$\begin{pmatrix} \tau_x \\ \tau_y \\ \tau_z \end{pmatrix} = \begin{pmatrix} I_{xx} & I_{xy} & I_{xz} \\ I_{yx} & I_{yy} & I_{yz} \\ I_{zx} & I_{zy} & I_{zz} \end{pmatrix} \begin{pmatrix} \alpha_x \\ \alpha_y \\ \alpha_z \end{pmatrix}$$

$$\tau_z = I_{zx}\alpha_x + I_{zy}\alpha_y + I_{zz}\alpha_z$$

Torque and angular acceleration may not be in the same direction!!!.

Consequence of off-diagonal terms in Moment of Inertia matrix



$$L_z = I_{zx}\omega_x + I_{zy}\omega_y + I_{zz}\omega_z$$

$$\tau_z = I_{zx}\alpha_x + I_{xy}\alpha_y + I_{zz}\alpha_z$$

Balanced Wheel



Consequence of off-diagonal terms in Moment of Inertia Matrix



$$L_z = I_{zx}\omega_x + I_{zy}\omega_y + I_{zz}\omega_z$$

$$\tau_z = I_{zx}\alpha_x + I_{zy}\alpha_y + I_{zz}\alpha_z$$

So, the off-diagonal terms

⇒ any attempt to rotate
the body by applying a
torque about a given axis
will not result in a
rotation about just that
axis, but there will be
rotation around the other
axes as well.

Off-diagonal elements represent **coupling** of rotation about one axis with other!!

$$L_{i} = L_{f}$$

$$I_{i} \omega_{i} = I_{f} \omega_{f}$$

Arms Extended

High Moment of inertia (I) Low Angular Velocity (ω)



Arms Tucked In

Low Moment of inertia (I) High Angular Velocity (ω)



Is Moment of Inertia a Scalar? Vector?

Or ????

Tensor

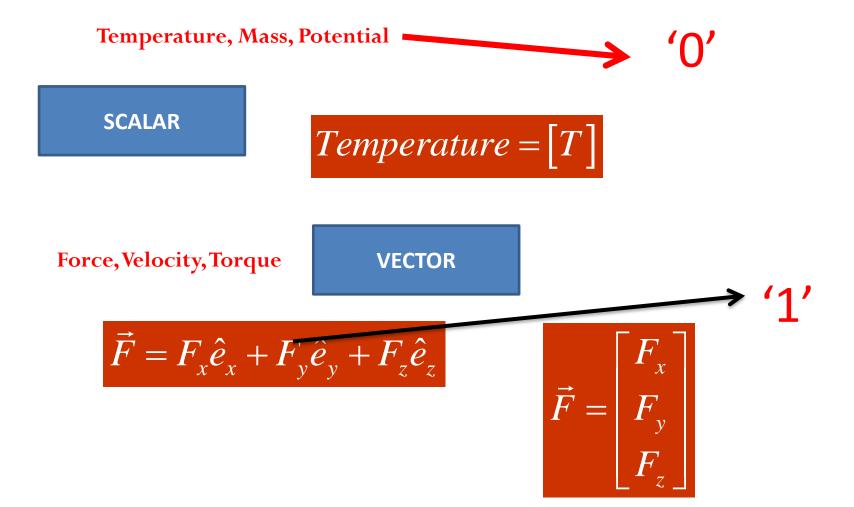
Tensor

Tensor represents a physical entity which may be characterized by magnitude and multiple directions simultaneously.

The rank of a tensor is defined by the number of directionality required to describe a component of it.

Tensor

The rank of a tensor is defined by the number of directionality required to describe a component of it.



Tensor

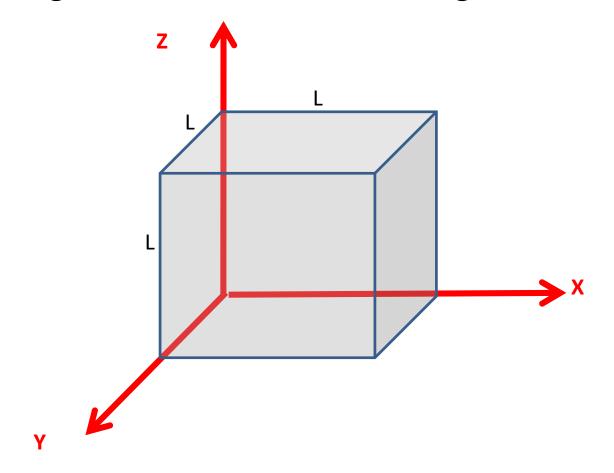
$$\tau_z = I_{zx}\alpha_x + I_{zy}\alpha_y + I_{zz}\alpha_z$$

Torque applied in z – direction need not guarantee angular acceleration only in x-direction, rather it can have y-component, x-component as well. This is due to coupling between two directions (Z-X, Z-Yetc).

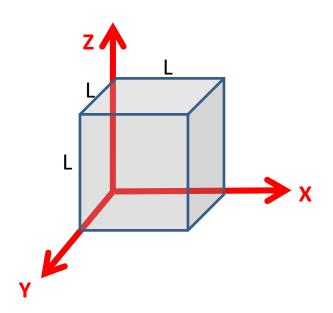
The rank of a tensor is defined by the number of directionality required to describe a component of it.

Rank 2 Tensor represents a physical entity which may be characterized by magnitude and bi-directionality.

Find the Moment of Inertia tensor of a cube with Side length L and mass M, with co-ordinate axis parallel to the edges of the cube and the origin at a corner.



$$\begin{bmatrix} I \end{bmatrix} = \begin{pmatrix} \int (y^2 + z^2)dm & -\int xydm & -\int zxdm \\ -\int xydm & \int (z^2 + x^2)dm & -\int yzdm \\ -\int zxdm & -\int yzdm & \int (x^2 + y^2)dm \end{pmatrix}$$



$$\int (y^2 + z^2) dm = \iiint (y^2 + z^2) \rho dx dy dz$$

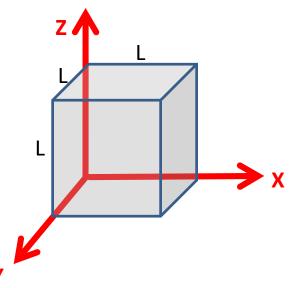
$$\iiint (y^2 + z^2) \rho dx dy dz = \rho L \int_0^L \int_0^L (y^2 + z^2) dy dz$$

$$\rho L \int_{0}^{L} \int_{0}^{L} \left(y^{2} + z^{2} \right) dy dz = \rho L \int_{0}^{L} \left(\frac{L^{3}}{3} + z^{2} L \right) dz$$

$$\int (y^2 + z^2) dm = \frac{2}{3} L^5 \rho = \frac{2}{3} ML^2$$

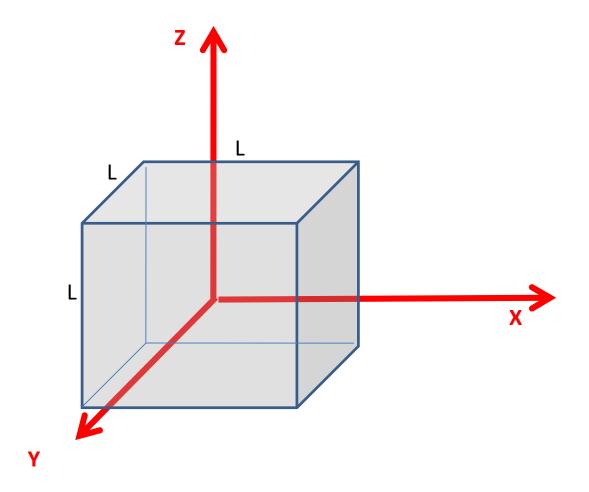


$$-\int (xy)dm = -\frac{ML^2}{4}$$



$$\begin{bmatrix} I \end{bmatrix} = ML^{2} \begin{pmatrix} 2/3 & -1/4 & -1/4 \\ -1/4 & 2/3 & -1/4 \\ -1/4 & -1/4 & 2/3 \end{pmatrix}$$





$$\int (y^2 + z^2)dm = \iiint (y^2 + z^2)\rho dxdydz$$

$$\iiint (y^2 + z^2) \rho dx dy dz = \rho L \int_{-L/2}^{L/2} \int_{-L/2}^{L/2} (y^2 + z^2) dy dz$$

$$\rho L \int_{-L/2}^{L/2} \int_{-L/2}^{L/2} \left(y^2 + z^2\right) dy dz = \rho L \int_{-L/2}^{L/2} \left(\frac{L^3}{12} + z^2 L\right) dz$$

$$\int \left(y^2 + z^2\right) dm = \frac{1}{6}ML^2$$

$$-\int (xy)dm = -\int xy\rho dxdydz$$

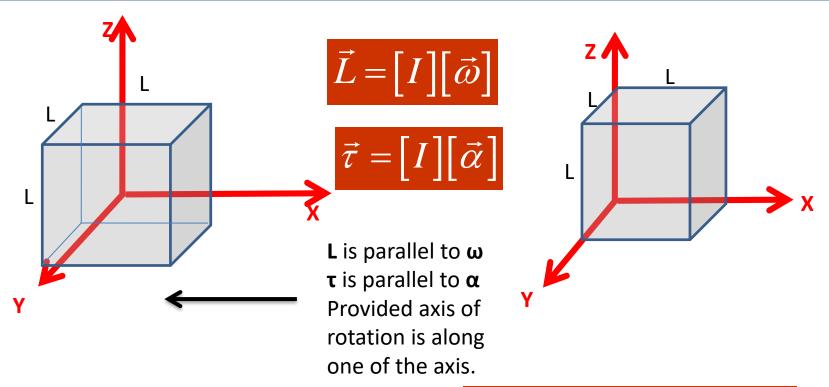
$$-\int (xy)dm = 0$$

$$\int (xy)dm = 0$$

$$\int (y^2 + z^2)dm = \frac{1}{6}ML^2$$

$$\begin{bmatrix} I \end{bmatrix} = ML^2 \begin{pmatrix} 1/6 & 0 & 0 \\ 0 & 1/6 & 0 \\ 0 & 0 & 1/6 \end{pmatrix}$$

Rotation of a cube along 2 different Axis



$$[I] = ML^2 \begin{pmatrix} 1/6 & 0 & 0 \\ 0 & 1/6 & 0 \\ 0 & 0 & 1/6 \end{pmatrix}$$

$$\begin{bmatrix} I \end{bmatrix} = ML^2 \begin{pmatrix} 2/3 & -1/4 & -1/4 \\ -1/4 & 2/3 & -1/4 \\ -1/4 & -1/4 & 2/3 \end{pmatrix}$$

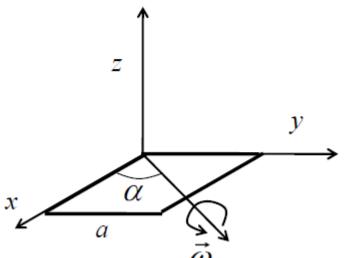
Important Observations

- Choosing a rotation axis at the center of the cube made the Moment of Inertia matrix diagonal
- Torque for rotation is given along a particular axis

L is parallel to ω τ is parallel to α What happens if a torque is applied off-axis?

Consider rotation of a square plate of side a and mass M about an axis in the plane of the plate and making an angle α with the x-axis.

- (a) What is the angular momentum **L** about the origin?
- (b) For what angle L and ω becomes parallel?
- (c) For square plate when the moment of inertia tensor becomes diagonal?



(a) What is the angular momentum L about the origin?

$$ec{L} = igl[I] igl[ec{arphi}igr]$$

$$\vec{L} = [I][\vec{\omega}]$$

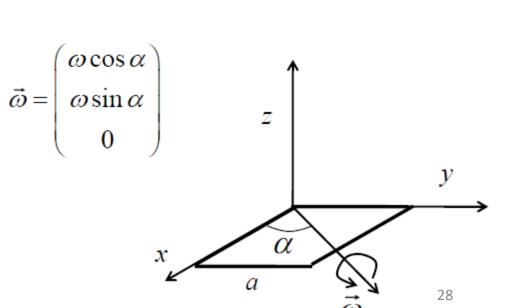
$$[I] = \begin{bmatrix} \int (y^2 + z^2)dm & -\int xydm & -\int zxdm \\ -\int xydm & \int (z^2 + x^2)dm & -\int yzdm \\ -\int zxdm & -\int yzdm & \int (x^2 + y^2)dm \end{bmatrix}$$

$$I_{xx} = \int_{x=0}^{a} \int_{y=0}^{a} \sigma y^{2} dx dy = \frac{1}{3} \sigma a^{4}$$

$$I_{xy} = -\int_{x=0}^{a} \int_{y=0}^{a} \sigma xy dx dy = -\frac{1}{4} \sigma a^{4}$$

$$\vec{\omega} = \begin{pmatrix} \omega \cos \alpha \\ \omega \sin \alpha \\ 0 \end{pmatrix}$$

$$I_{zz} = \frac{2}{3} Ma^{2}$$



(a) What is the angular moment L about the origin?

$$I_{xx} = \int_{x=0}^{a} \int_{y=0}^{a} \sigma y^2 dx dy = \frac{1}{3} \sigma a^4$$

$$I_{xy} = -\int_{x=0}^{a} \int_{y=0}^{a} \sigma xy dx dy = -\frac{1}{4} \sigma a^4$$

$$I_{zz} = \frac{2}{3}Ma^2$$

$$\vec{L} = \begin{pmatrix} Ma^2/3 & -Ma^2/4 & 0 \\ -Ma^2/4 & Ma^2/3 & 0 \\ 0 & 0 & 2Ma^2/2 \end{pmatrix}$$

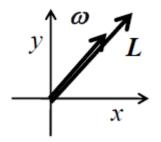
$$\vec{\omega} = \begin{pmatrix} \omega \cos \alpha \\ \omega \sin \alpha \\ 0 \end{pmatrix}$$

$$\vec{L} = \begin{pmatrix} Ma^{2}/3 & -Ma^{2}/4 & 0\\ -Ma^{2}/4 & Ma^{2}/3 & 0\\ 0 & 0 & 2Ma^{2}/3 \end{pmatrix} \begin{pmatrix} \omega \cos \alpha \\ \omega \sin \alpha \\ 0 \end{pmatrix} = \begin{pmatrix} Ma^{2}\omega \left(\frac{1}{3}\cos \alpha - \frac{1}{4}\sin \alpha\right) \\ Ma^{2}\omega \left(-\frac{1}{4}\cos \alpha + \frac{1}{3}\sin \alpha\right) \\ 0 \end{pmatrix}$$

(b) For what angle L and ω becomes parallel?

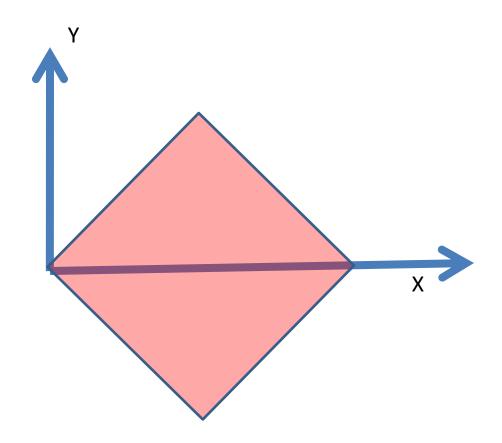
For
$$\alpha = 45^{\circ}$$
,

$$\vec{L} = \left(\frac{1}{12\sqrt{2}}Ma^2\omega, \frac{1}{12\sqrt{2}}Ma^2\omega, 0\right) \text{ and } \vec{\omega} = \begin{pmatrix} \omega/\sqrt{2} \\ \omega/\sqrt{2} \\ 0 \end{pmatrix}$$

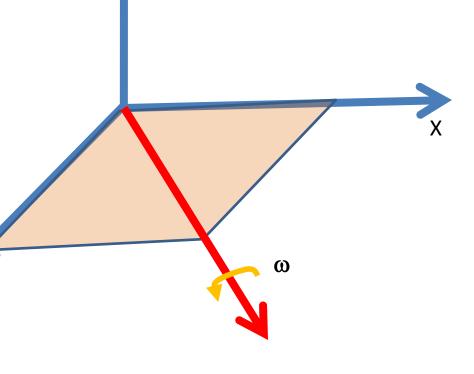


$$\vec{L} = [I][\vec{\omega}] = \lambda \vec{\omega}$$

(c) for the square plate when the moment of inertia tensor becomes diagonal?



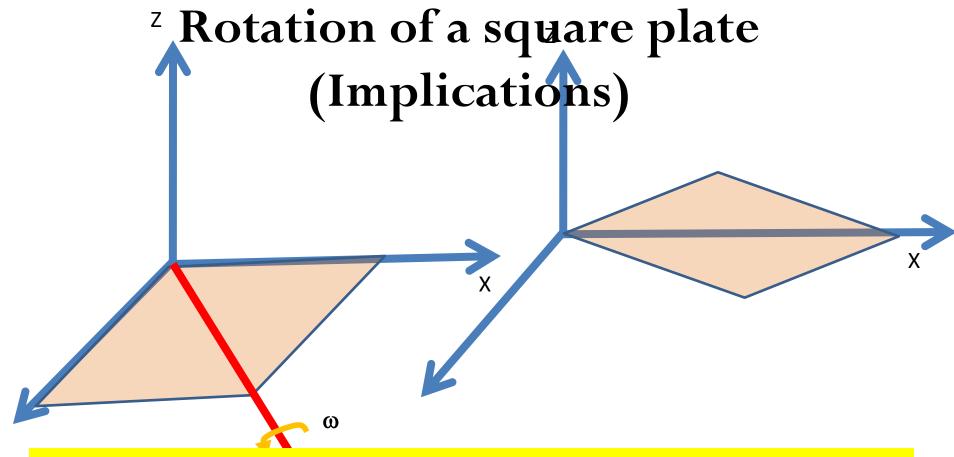
Rotation of a square plate (Implications)



L and
$$\omega$$
 are parallel!

$$\begin{bmatrix} I \end{bmatrix} = \begin{pmatrix} I_{xx} & I_{xy} & I_{xz} \\ I_{yx} & I_{yy} & I_{yz} \\ I_{zx} & I_{zy} & I_{zz} \end{pmatrix}$$

$$\vec{L} = [I][\vec{\omega}]$$



Use of symmetry will ensure diagonal Moment of Inertia tensor

$$[I] = \begin{pmatrix} I_{xx} & 0 & 0 \\ 0 & I_{yy} & 0 \\ 0 & 0 & I_{zz} \end{pmatrix}$$

$$L_{x} = I_{xx}\omega_{x}$$

$$\tau_{x} = I_{xx}\alpha_{x}$$

Principal Axis

$$\begin{bmatrix} I \end{bmatrix} = \begin{pmatrix} I_{xx} & I_{xy} & I_{xz} \\ I_{yx} & I_{yy} & I_{yz} \\ I_{zx} & I_{zy} & I_{zz} \end{pmatrix}$$
 Cun

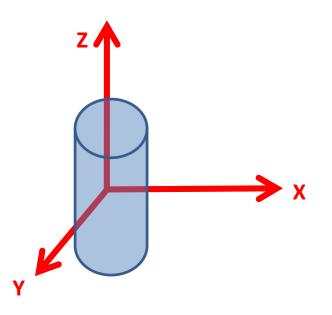
Cumbersome!

Principal axes are the orthogonal axes for Which [I] is diagonal

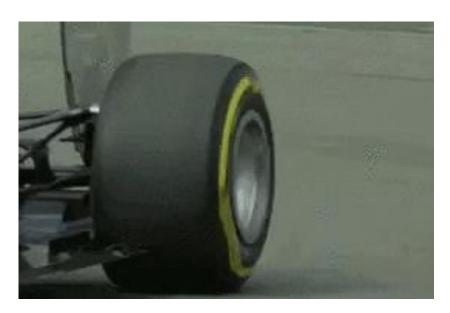
$$[I] = \begin{pmatrix} I_{xx} & 0 & 0 \\ 0 & I_{yy} & 0 \\ 0 & 0 & I_{zz} \end{pmatrix}$$

$$L_{x} = I_{xx}\omega_{x} L_{y} = I_{yy}\omega_{y}$$

$$L_{z} = I_{zz}\omega_{z}$$



Non-zero Product of Inertia

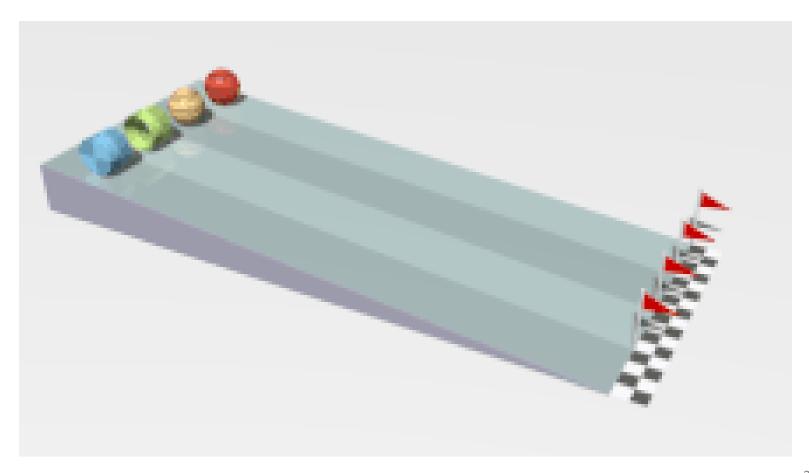






Importance of moment of inertia Winter Olympics, 2006 Turin, Italy

Importance of moment of inertia

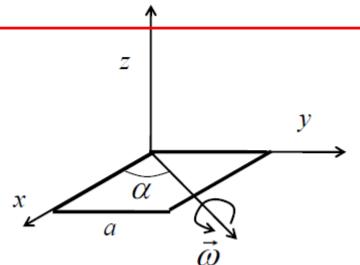


Rotation of a square plate

Consider rotation of a square plate of side a and mass M about an axis in the plane of the plate and making an angle α with the x-axis.

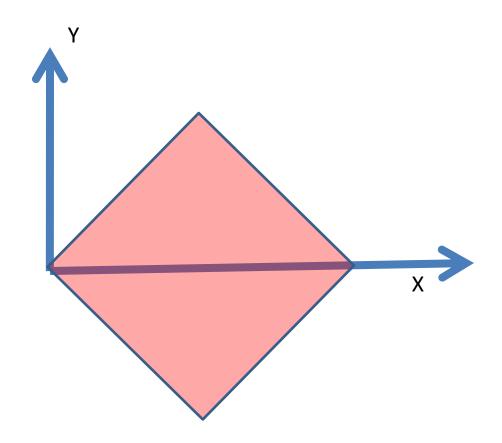
- (a) What is the angular momentum L about the origin?
- (b) For what angle L and ω becomes parallel?

(c) For square plate when the moment of inertia tensor becomes diagonal?

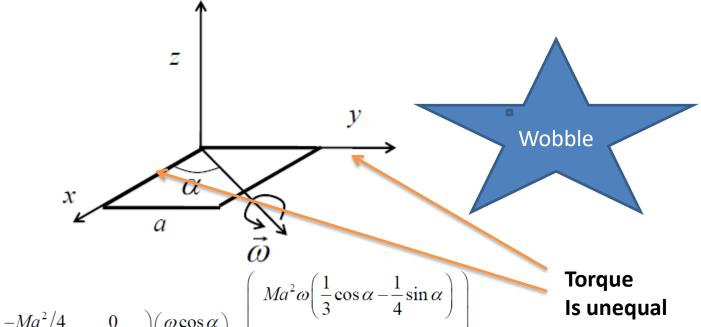


Rotation of a square plate

(c) for the square plate when the moment of inertia tensor becomes diagonal?



Concept behind the problem



$$\vec{L} = \begin{pmatrix} Ma^{2}/3 & -Ma^{2}/4 & 0\\ -Ma^{2}/4 & Ma^{2}/3 & 0\\ 0 & 0 & 2Ma^{2}/3 \end{pmatrix} \begin{pmatrix} \omega \cos \alpha \\ \omega \sin \alpha \\ 0 \end{pmatrix} = \begin{pmatrix} Ma^{2}\omega \left(\frac{1}{3}\cos \alpha - \frac{1}{4}\sin \alpha\right) \\ Ma^{2}\omega \left(-\frac{1}{4}\cos \alpha + \frac{1}{3}\sin \alpha\right) \\ 0 \end{pmatrix}$$

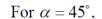
Along X and Y axis

L is not parallel to ω τ is not parallel to α

$$\boldsymbol{\tau} = \begin{bmatrix} Ma^2 \left(\frac{1}{3}cos\alpha - \frac{1}{4}sin\alpha\right)\dot{\alpha} \\ Ma^2 \left(-\frac{1}{4}cos\alpha + \frac{1}{3}sin\alpha\right)\dot{\alpha} \\ 0 \end{bmatrix}_{40}$$



Rotation at 45 degrees



$$\vec{L} = \begin{bmatrix} \frac{1}{12\sqrt{2}} Ma^2 v, \frac{1}{12\sqrt{2}} Ma^2 \omega, 0 \end{bmatrix} \text{ and } \vec{\omega} = \begin{bmatrix} \omega/\sqrt{2} \\ \omega/\sqrt{2} \\ 0 \end{bmatrix}$$

$$\vec{L} = [I][\vec{\omega}] = \lambda \vec{\omega}$$

L and ω are parallel!

$$\boldsymbol{\tau} = \begin{bmatrix} Ma^2 \left(\frac{1}{12\sqrt{2}}\right) \dot{\alpha} \\ Ma^2 \left(\frac{1}{12\sqrt{2}}\right) \dot{\alpha} \\ 0 \end{bmatrix}$$

Torque
Is equal
Along
X and Y axis

Does the Plate Wobble?

Answer is NO

L is parallel to ω τ is parallel to α

$$\boldsymbol{\tau} = \begin{bmatrix} Ma^2 \left(\frac{1}{12\sqrt{2}}\right) \dot{\alpha} \\ Ma^2 \left(\frac{1}{12\sqrt{2}}\right) \dot{\alpha} \\ 0 \end{bmatrix}$$

TORQUE IS BALANCED: Component of torque are equal in X AND Y directions

Note!

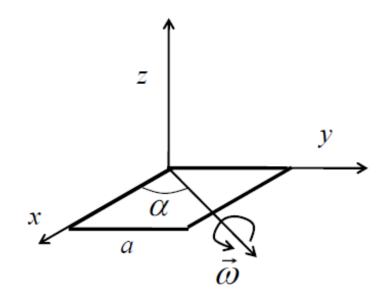
• Whenever ω is parallel to L, choosing the corresponding direction of rotation as co-ordinate axis will be the <u>principal axes</u>.

For simple geometries it is easy to find by intuition but not for any general case

How to find diagonalized MI tensor??

Mathematical approach!

$$\begin{bmatrix} I \end{bmatrix} = \begin{pmatrix} I_{xx} & I_{xy} & I_{xz} \\ I_{yx} & I_{yy} & I_{yz} \\ I_{zx} & I_{zy} & I_{zz} \end{pmatrix}$$



$$\vec{L} = [I][\vec{\omega}]$$

$$[I][\vec{\omega}] = \lambda \vec{\omega}$$

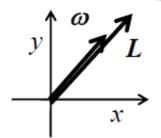
We can write like that when L and ω are parallel

Rotation of a square plate

(b) For what angle L and ω becomes parallel?

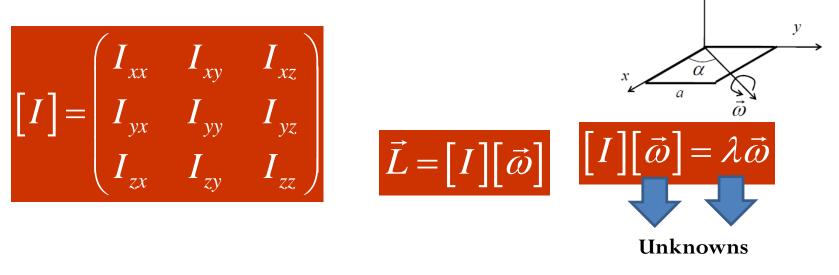
For
$$\alpha = 45^{\circ}$$
,

$$\vec{L} = \left(\frac{1}{12\sqrt{2}} Ma^2 \omega , 0 \right) \text{ and } \vec{\omega} = \begin{bmatrix} \omega/\sqrt{2} \\ \omega/\sqrt{2} \\ 0 \end{bmatrix}$$



$$\vec{L} = [I][\vec{\omega}] = \lambda \vec{\omega}$$

$$\begin{bmatrix} I \end{bmatrix} = \begin{pmatrix} I_{xx} & I_{xy} & I_{xz} \\ I_{yx} & I_{yy} & I_{yz} \\ I_{zx} & I_{zy} & I_{zz} \end{pmatrix}$$



Step: 1

Find the axis of rotation where L and ω are parallel.

WE IMPOSE THE CONDITION, TO MAKE L AND ω PARALLEL

$$[I][\vec{\omega}] = \lambda \vec{\omega}$$

Find the unknowns ω and λ

$$[I][\vec{\omega}] = \lambda \vec{\omega}$$

$$\begin{pmatrix}
I_{xx} & I_{xy} & I_{xz} \\
I_{yx} & I_{yy} & I_{yz} \\
I_{zx} & I_{zy} & I_{zz}
\end{pmatrix}
\begin{pmatrix}
\omega_{x} \\
\omega_{y} \\
\omega_{z}
\end{pmatrix} = \lambda
\begin{pmatrix}
\omega_{x} \\
\omega_{y} \\
\omega_{z}
\end{pmatrix}$$

Insert a unit matrix

$$\begin{pmatrix} I_{xx} & I_{xy} & I_{xz} \\ I_{yx} & I_{yy} & I_{yz} \\ I_{zx} & I_{zy} & I_{zz} \end{pmatrix} \begin{pmatrix} \omega_x \\ \omega_y \\ \omega_z \end{pmatrix} = \begin{pmatrix} \lambda & 0 & 0 \\ 0 & \lambda & 0 \\ 0 & 0 & \lambda \end{pmatrix} \begin{pmatrix} \omega_x \\ \omega_y \\ \omega_z \end{pmatrix}$$

$$[MI] - \lambda [1] = 0$$

$$\begin{pmatrix} I_{xx} - \lambda & I_{xy} & I_{xz} \\ I_{yx} & I_{yy} - \lambda & I_{yz} \\ I_{zx} & I_{zy} & I_{zz} - \lambda \end{pmatrix} \begin{pmatrix} \omega_x \\ \omega_y \\ \omega_z \end{pmatrix} = 0$$

Theorem: If [A][x]=0, then [A] is non-invertible. This implies A^{-1} does not exist Hence, |A|=0.

MA102 Mathematics - II

MA102 Mathematics - II 3-1-0-8 Pre-requisites: nil

Linear Algebra: Vector spaces (over the field of real and complex numbers). Systems of linear equations and their solutions.

Matrices, determinants, rank and inverse. Linear transformations. Range space and rank, null space and nullity Eigenvalues and eigenvectors. Similarity transformations. Diagonalization of Hermitian matrices. Billinear and guadratic forms.

$$\begin{bmatrix} I_{xx} - \lambda & I_{xy} & I_{xz} \\ I_{yx} & I_{yy} - \lambda & I_{yz} \\ I_{zx} & I_{zy} & I_{zz} - \lambda \end{bmatrix} \begin{pmatrix} \omega_x \\ \omega_y \\ \omega_z \end{pmatrix} = 0$$

$$\begin{vmatrix} I_{xx} - \lambda & I_{xy} & I_{xz} \\ I_{yx} & I_{yy} - \lambda & I_{yz} \\ I_{zx} & I_{zy} & I_{zz} - \lambda \end{vmatrix} = 0$$

Characteristic Equation

$$\begin{vmatrix} I_{xx} - \lambda & I_{xy} & I_{xz} \\ I_{yx} & I_{yy} - \lambda & I_{yz} \\ I_{zx} & I_{zy} & I_{zz} - \lambda \end{vmatrix} = 0$$

Solving characteristic equation result in $\lambda_1, \lambda_2, \lambda_3$

$$\begin{pmatrix} I_{xx} - \lambda & I_{xy} & I_{xz} \\ I_{yx} & I_{yy} - \lambda & I_{yz} \\ I_{zx} & I_{zy} & I_{zz} - \lambda \end{pmatrix} \begin{pmatrix} \omega_x \\ \omega_y \\ \omega_z \end{pmatrix} = 0$$
 Substitute
$$\lambda_1, \lambda_2, \lambda_3 \text{ separately and solve for } \omega_s$$

Note!

• $\lambda_{1,} \lambda_{2}$, λ_{3} are called the Eigen values which satisfies the equation

$$[I][\vec{\omega}] = \lambda \vec{\omega}$$

 \bullet [ω]'s are called the Eigen vectors.

• For each Eigen values, Eigen vector can be found.

Step2: How to find diagonalized MI tensor??

Diagonalization Theorem,

$$D = \begin{pmatrix} \lambda_1 & 0 & 0 \\ 0 & \lambda_2 & 0 \\ 0 & 0 & \lambda_3 \end{pmatrix}$$

Diagonalization ensures the rotation axis is along the coordinate axis (Principal axis)

(Will be taught in MA102)

MA102 Mathematics - II

Mathematics - II MA102 3-1-0-8 Pre-requisites: nil

Linear Algebra: Vector spaces (over the field of real and complex numbers). Systems of linear equations and their solutions Matrices, determinants, rank and inverse. Linear transformations. Range space and rank, null space and nullity Eigenvalues

and eigenvectors. Similarity transformations. Diagonalization of Hermitian matrices. Bilinear and quadratic forms.

Find principal axis of a rigid body whose moment of inertia

tensor is given as

$$I = \begin{bmatrix} 2 & -1 \\ -1 & 2 \end{bmatrix}$$

$$[I][\vec{\omega}] = \lambda \vec{\omega}$$

$$\begin{vmatrix} 2 - \lambda & -1 \\ -1 & 2 - \lambda \end{vmatrix} = 0$$

Characteristic Equation

$$\lambda^2 - 4\lambda + 3 = 0$$

$$\lambda_1 = 1, \lambda_2 = 3,$$

Find Principal axis of a rigid body

Use λ_1 in eigenvalue expression:

$$\omega_{x} = \omega_{y} = a$$

$$\omega_{x} = \omega_{y} = a$$

Eigenvalue equation corresponding to λ_1 .

Eigenvector corresponding to
$$\lambda_{1:}$$

$$\begin{pmatrix} 2-1 & -1 \\ -1 & 2-1 \end{pmatrix} \begin{pmatrix} \omega_x \\ \omega_y \end{pmatrix} = 0$$

$$[I][\vec{\omega}] = \lambda \vec{\omega}$$

$$[I][\omega] = \begin{bmatrix} 2 & -1 \\ -1 & 2 \end{bmatrix} \begin{bmatrix} a \\ a \end{bmatrix} = \begin{bmatrix} a \\ a \end{bmatrix}$$



Find Principal axis of a rigid body

Use λ_2 in eigenvalue $\begin{pmatrix} 2-3 & -1 \\ -1 & 2-3 \end{pmatrix} \begin{pmatrix} \omega_x \\ \omega_y \end{pmatrix} = 0$ expression:

$$\begin{pmatrix} 2-3 & -1 \\ -1 & 2-3 \end{pmatrix} \begin{pmatrix} \omega_x \\ \omega_y \end{pmatrix} = 0$$

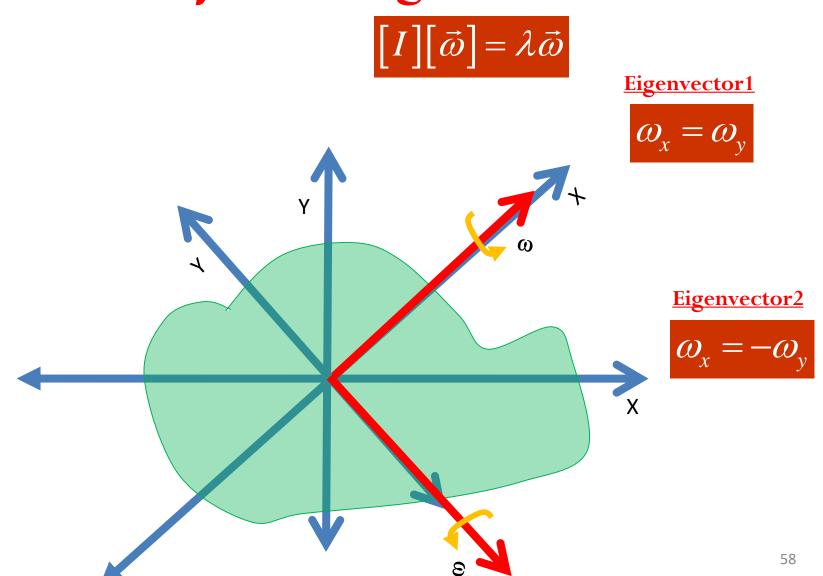
$$\omega_{x} = -\omega_{y}$$

Eigenvector 1 corresponding to λ_2 : -1

$$\begin{bmatrix} 1 \\ -1 \end{bmatrix}$$

Verify orthogonality of eigenvectors

Finding Principal axis of a rigid body from eigenvectors



Moment of Inertia matrix for principal axis

$$\lambda_1 = 1, \lambda_2 = 3,$$

$$I = \begin{bmatrix} 1 & 0 \\ 0 & 3 \end{bmatrix}$$

$$L = \begin{bmatrix} 1 & 0 \\ 0 & 3 \end{bmatrix} \begin{bmatrix} \omega_x \\ 0 \end{bmatrix} = 1 \begin{bmatrix} \omega_x \\ 0 \end{bmatrix}$$

$$L = \begin{bmatrix} 1 & 0 \\ 0 & 3 \end{bmatrix} \begin{bmatrix} 0 \\ \omega_y \end{bmatrix} = 3 \begin{bmatrix} 0 \\ \omega_y \end{bmatrix}$$

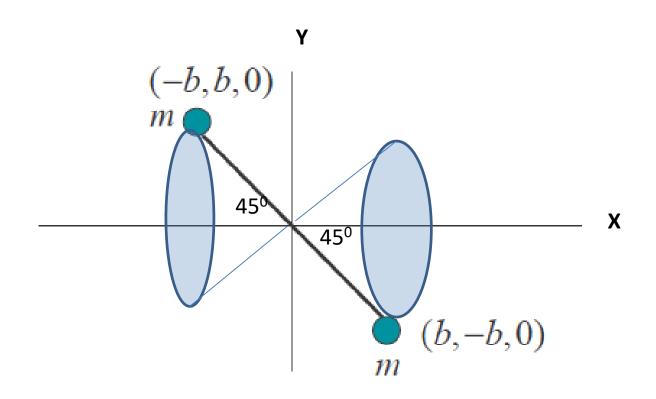
Another example.....

ROTATING DUMBBELL

A dumb-bell is rotated along X and Y axis as shown in figure below

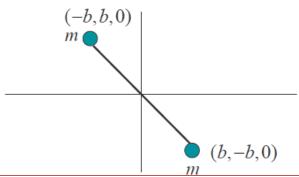
A. Find the Moment of Inertia Tensor

B. Find the principal axis corresponding to it



A. Moment of Inertia Tensor

$$[I] = \begin{pmatrix} \int (y^2 + z^2)dm & -\int xydm & -\int zxdm \\ -\int xydm & \int (z^2 + x^2)dm & -\int yzdm \\ -\int zxdm & -\int yzdm & \int (x^2 + y^2)dm \end{pmatrix}$$



$$\mathbf{I} = m \begin{bmatrix} 2b^2 & 2b^2 & 0 \\ 2b^2 & 2b^2 & 0 \\ 0 & 0 & 4b^2 \end{bmatrix} = 2b^2 m \begin{bmatrix} 1 & 1 & 0 \\ 1 & 1 & 0 \\ 0 & 0 & 2 \end{bmatrix}$$

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Finding the Eigenvalues

$$\begin{vmatrix} I_{xx} - \lambda & I_{xy} & I_{xz} \\ I_{yx} & I_{yy} - \lambda & I_{yz} \\ I_{zx} & I_{zy} & I_{zz} - \lambda \end{vmatrix} = 0$$

$$\begin{vmatrix} 2mb^2 - \lambda & 2mb^2 & 0 \\ 2mb^2 & 2mb^2 - \lambda & 0 \\ 0 & 0 & 4mb^2 - \lambda \end{vmatrix} = 0$$

Eigenvalues

$$\begin{vmatrix} 2mb^2 - \lambda & 2mb^2 & 0 \\ 2mb^2 & 2mb^2 - \lambda & 0 \\ 0 & 0 & 4mb^2 - \lambda \end{vmatrix} = 0$$

$$\left[(4mb^2 - \lambda) \left[\left(2mb^2 - \lambda \right)^2 - \left(2mb^2 \right)^2 \right] = 0$$

$$\left[(4mb^2 - \lambda) \left[\lambda^2 - 4mb^2 \lambda \right] = 0 \right]$$

$$(4mb^2 - \lambda)(\lambda)(\lambda - 4mb^2) = 0$$

$$\lambda_1 = \lambda_2 = 4mb^2, \lambda_3 = 0$$

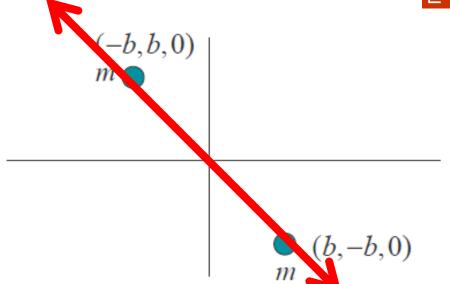
Principal axis and eigenvectors

for
$$\lambda_3 = 0$$

$$\omega_x = -\omega_y$$
$$\omega_z = 0$$







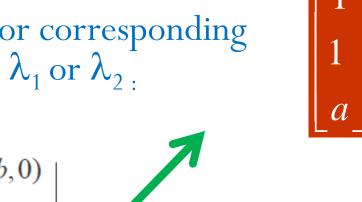
Principal axis and eigenvectors

for
$$\lambda_1 = \lambda_2 = 4mb^2$$

$$\omega_{x} = \omega_{y}$$

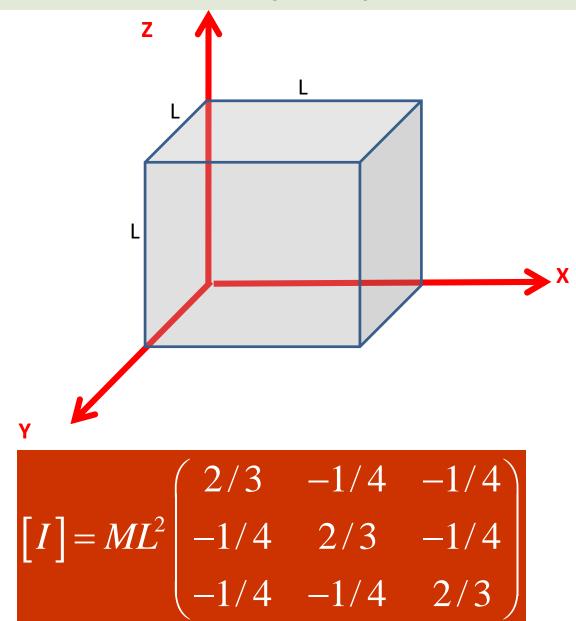
$$\omega_{z} = Anything$$

Eigenvector corresponding to λ_1 or λ_2 .

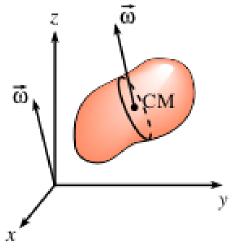


Verify orthogonality of eigenvectors

HW: Find principal axis



Parallel-axis Theorem



CM rotates around the origin with the same angular velocity at which the body rotates around the CM

$$\mathbf{L} = M\mathbf{R} \times (\boldsymbol{\omega} \times \mathbf{R}) + \int \mathbf{r}' \times (\boldsymbol{\omega} \times \mathbf{r}') \, dm$$

Writing the double cross products in matrix form, we get

$$\begin{pmatrix} L_1 \\ L_2 \\ L_3 \end{pmatrix} = M \begin{pmatrix} Y^2 + Z^2 & -XY & -ZX \\ -XY & Z^2 + X^2 & -YZ \\ -ZX & -YZ & X^2 + Y^2 \end{pmatrix} \begin{pmatrix} \omega_1 \\ \omega_2 \\ \omega_3 \end{pmatrix}$$

$$+ \begin{pmatrix} \int (y'^2 + z'^2) & -\int x'y' & -\int z'x' \\ -\int x'y' & \int (z'^2 + x'^2) & -\int y'z' \\ -\int z'x' & -\int y'z' & \int (x'^2 + y'^2) \end{pmatrix} \begin{pmatrix} \omega_1 \\ \omega_2 \\ \omega_3 \end{pmatrix}$$

$$\equiv (\mathbf{I}_{R} + \mathbf{I}_{CM})\boldsymbol{\omega}.$$

Kinetic energy:
$$T = \frac{1}{2} \boldsymbol{\omega} \cdot (\boldsymbol{I}_R + \boldsymbol{I}_{CM}) \boldsymbol{\omega} = \frac{1}{2} \boldsymbol{\omega} \cdot \boldsymbol{L}$$

Few scattered thoughts about rigid body rotation

- (1) <u>Co-ordinate system</u> (Choice of axis) defines MI tensor. <u>Axis of rotation</u> is immaterial in defining MI tensor.
- (2) Different co-ordinate axis will result in different MI tensor even if <u>rotation is physically along same axis!!!</u>

(3) Different co-ordinate system will only change mathematics! **Rotation dynamics will be same**!

(4) Rotation is concrete (Real), but co-ordinate choice is abstract.

Few scattered thoughts about rigid body rotation

(5) If MI tensor is non-diagonal, L and ω <u>need not be</u> in same direction.

(6) Non-diagonal nature of MI tensor <u>does not</u> guarantee non-parallel nature of L and ω. (They can be in different direction)

(7) Rotation through one axis will induce coupling with other axis if MI tensor is non-diagonal: Bi-directional coupling. MI tensor is dyadic.

Few scattered thoughts about rigid body rotation

(8) If we choose ω axis properly, L and ω can be in same direction even if MI tensor is non-diagonal.

$$L = [I][\vec{\omega}] = \lambda \vec{\omega}$$

$$L_{x} = I_{xx}\omega_{x} + I_{xy}\omega_{y}$$

$$L_{y} = I_{xy}\omega_{x} + I_{yy}\omega_{y}$$
Different

(9) If Co-ordinate axis (Primed axis) is chosen along ω axis,

$$L_{x'} = I '_{xx} \omega_{x'}$$

$$ig|L = ig[Iig] ig[ec{arphi}ig] = \lambda ec{arphi}ig|$$

$$\begin{pmatrix} I_{xx} - \lambda & I_{xy} & I_{xz} \\ I_{yx} & I_{yy} - \lambda & I_{yz} \\ I_{zx} & I_{zy} & I_{zz} - \lambda \end{pmatrix} \begin{pmatrix} \omega_x \\ \omega_y \\ \omega_z \end{pmatrix} = 0$$

$$\begin{vmatrix} I_{xx} - \lambda & I_{xy} & I_{xz} \\ I_{yx} & I_{yy} - \lambda & I_{yz} \\ I_{zx} & I_{zy} & I_{zz} - \lambda \end{vmatrix} = 0$$

Characteristic Equation