

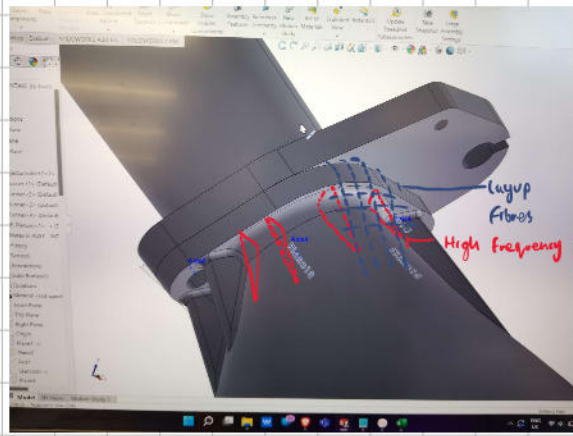


## Objectives?

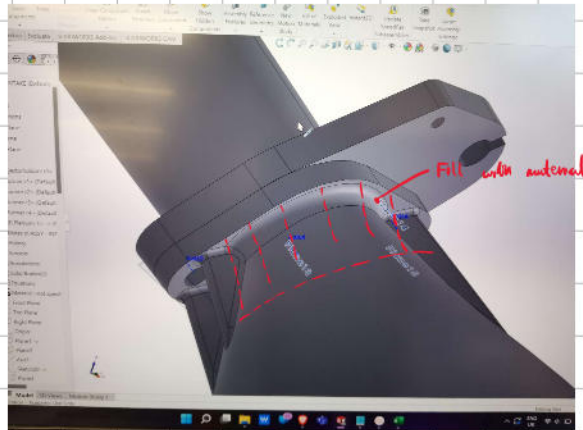
- 1, 3, 4 are MUST
- 7 will be useful - write some code (saves lots of effort)
- 2 and 5 - lower priority, 'would be nice to have'

Intake Design Review 1: 26/10/22

Arden-Make-Tusen

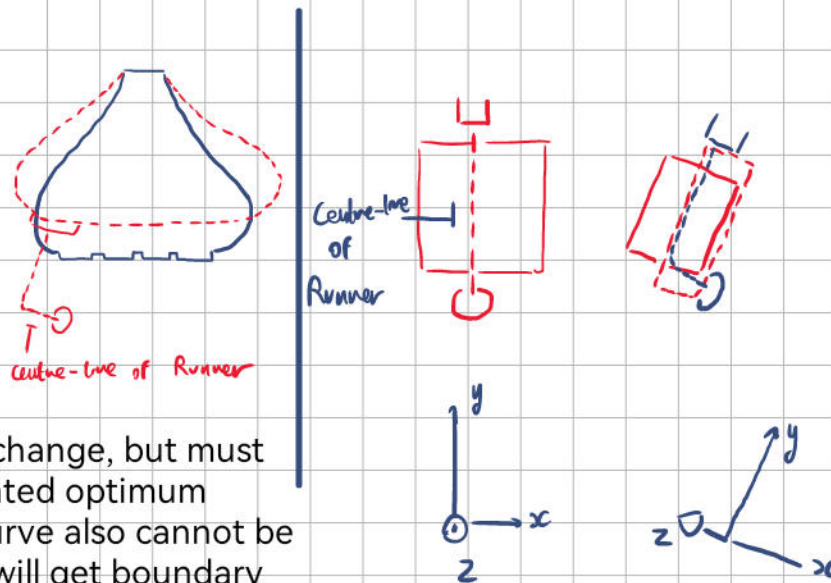


Due to bending moments at the neck of the plenum, and as 3D prints are weak in shear orthogonal to the plan of print layers - we must lay that up.



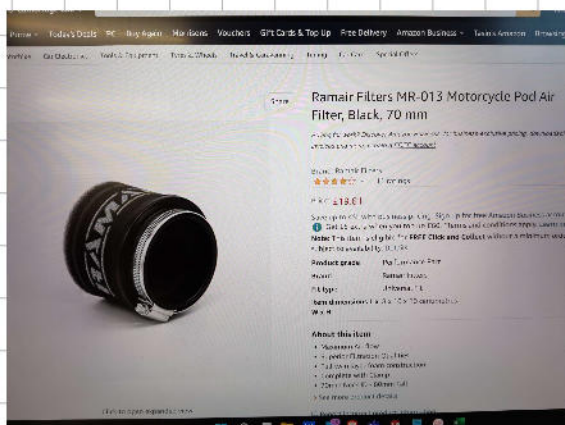
The layup of glass or carbon fibre provides a 'pre-tensed' load, which prevents fracture of the neck under bending.

(2)



Plenum volume can also change, but must remain around the simulated optimum volume of 750mm. The curve also cannot be too sharp, otherwise you will get boundary layer separation resulting in losses to the airflow going into the engine cylinders at the edges (1 and 4).

(3)

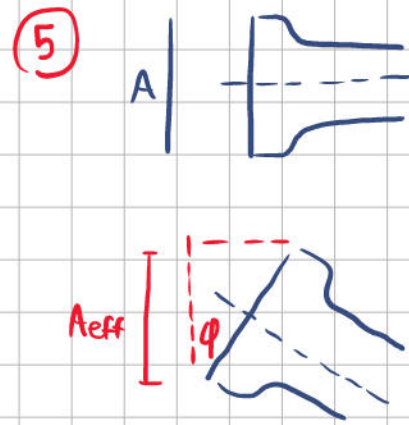
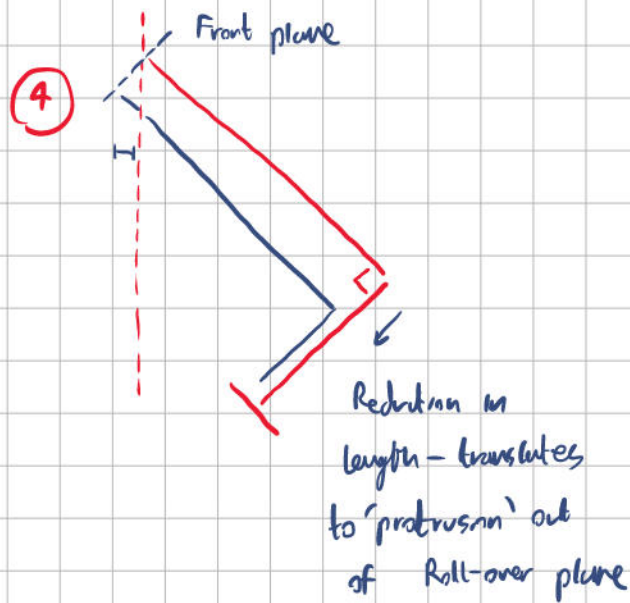


- 1 Rotation about z
- 2 Rotation about y
- 3 Requires small change in inlet 'hole' shell

New air filter: ~60mm ( $\pm 20$ mm) - not sure if product includes 'attachment' dimensions)

Current: ~150mm





$$A_{eff} = \cos(\phi) A$$

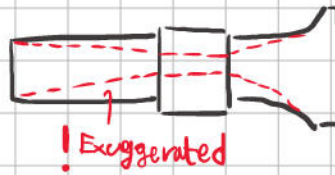
While 290mm is an upper bound for the runner length, this does include the curved path length of the injector holders as well. Overall then, the length limit is lower.

From the simulations, there is a greater drop-off at high end (high RPM) torque for longer lengths, so actually a shorter runner length is better.

Effective cross-sectional area facing in direct air flow is also an issue- try to keep this fixed, but avoid making the angle any further vertical.

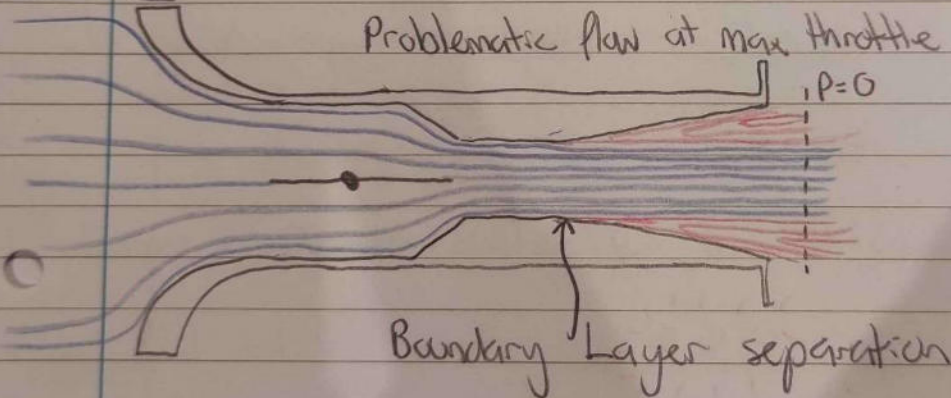
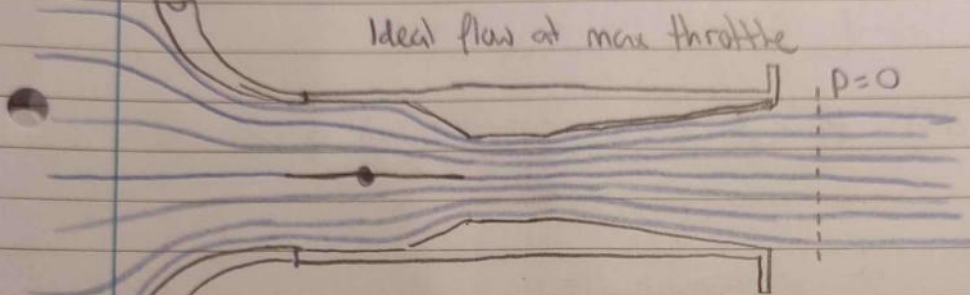
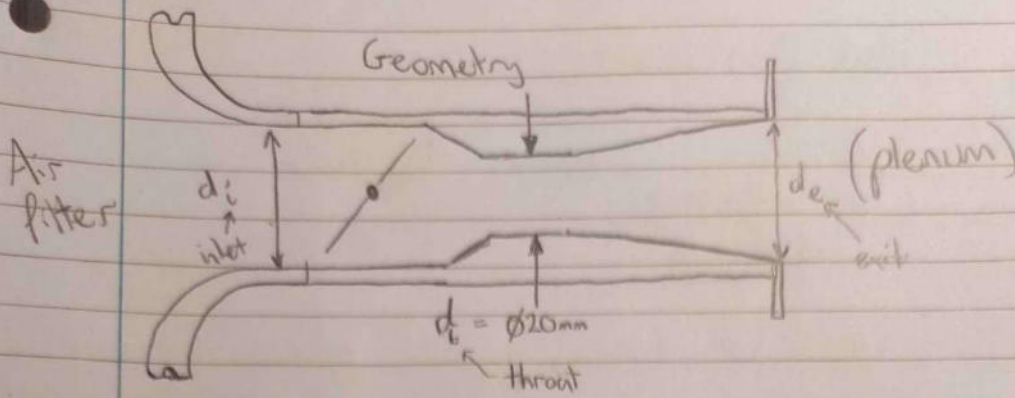
Relevant for 'ram' air effect.

## ⑥ Throttle body:



- ▶ Large, open mouth
- ▶ Narrows at restrictor
- ▶ low pressure gradient at diffusion region -

to prevent separation, results in back pressure. This back pressure reduces pressure gradient, reducing net inflow of air



$$V_t = V_i \times \frac{d_i^2}{d_t^2} \leftarrow \text{conservation of mass}$$

$$P_t + \frac{1}{2} \rho V_t^2 = P_i + \frac{1}{2} \rho V_i^2 \leftarrow \text{conservation of energy}$$

$\uparrow$  PE/unit volume       $\uparrow$  KE/unit volume

$$P_t = P_i - \frac{1}{2} \rho V_i^2 \left( \frac{d_i^2}{d_t^2} - 1 \right) \leftarrow \text{combine to show } P_t < P_i$$

★ doesn't tell you any values, just difference between them

ideal flow case:

$$V_e = V_i \quad (\text{assuming } d_i = d_e)$$
$$\therefore p + \frac{1}{2} \rho v^2 = \text{const}$$
$$\rightarrow P_e = P_i$$

we'll set datum pressure  $P_e = 0$  (in reality this is MAP)

$$\text{if } P_e = P_i = 0$$

and  $P_b < P_i$ ,  $P_b < 0$  all is well with the world

in problematic flow

$d_e = d_b$  !! (bernoulli only applies to blue streamlines so use that diameter)

$$P_e = P_b$$

but still  $P_b < P_i$  (still same upstream contraction)

$$\therefore 0 < P_i$$

Now we have a big high pressure zone in inlet blocking incoming flow.

This restricts the flow rate substantially

$$\dot{m}_{\text{ideal}} \gg \dot{m}_{\text{problematic}}$$

! CAN be shortened - as long as diffusion region is long and shallow





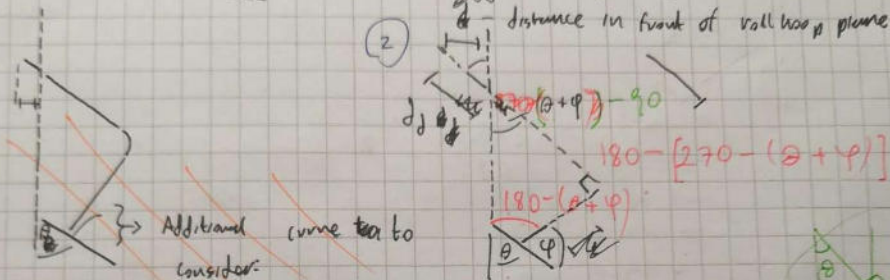
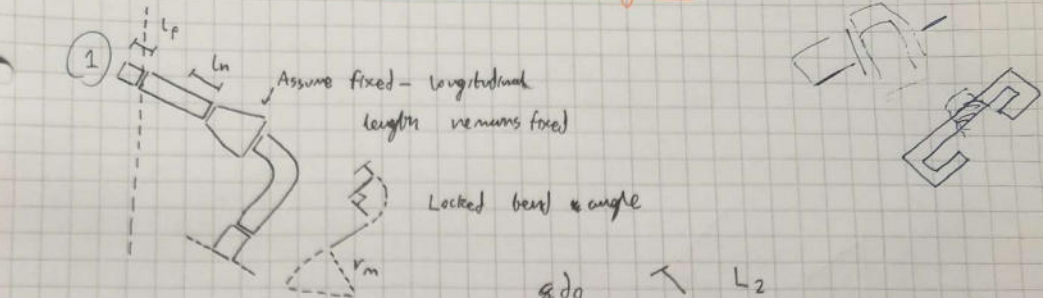
## ⑦ Parametrisation:

Consider some variable which is the horizontal overhang of the throttlebody an air filter in front of the rear roll hoop plane.

We can select a geometry, such that adjusting a variety of parameters: Runner straight section length, injector holer bend radius, Plenum longitudinal length etc... gives the following outputs: (1) Overhang in x in front of rollhoop (2) Vertical height of throttlebody mouth

i.e. If too low, defeats purpose of 'ram air' effect: but behind headrest

& Air Intake Geometry



① Turnable lengths  
/ fixed / locked



$$d_0 = \frac{L_1}{\tan(\theta + \phi)} + L_2 = \frac{L_1}{\tan(\theta + \phi)} + L_2$$

$$L_2 = (L_1 - d_0) \tan(\theta + \phi) = L_1$$

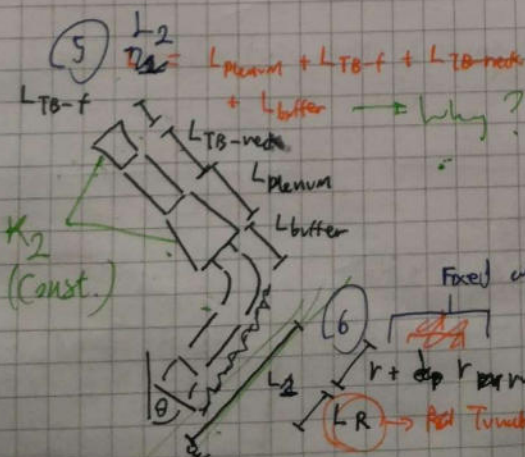
$d_0 = \sin(\theta + \phi) d_0$  → Mathematically correct but be careful!

$$d_0 = \frac{L_1}{\tan(\theta + \phi)} + L_2$$

$$d_0 = \sin(\theta + \phi) \left( L_2 + \frac{L_1}{\tan(\theta + \phi)} \right)$$

$$d_0 = \sin(\theta + \phi) L_2 + \frac{\cos(\theta + \phi) L_1}{\cos(\theta + \phi)}$$

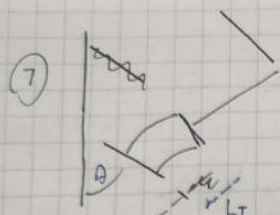
Some solution s.t.  $d_0 \leq 0$   $\frac{\sin^2(\theta + \phi)}{\cos(\theta + \phi)}$



Fixed as 90°

r + d p r runner

L R → Red Turnable



- fixed by position on cranklike manifold gear
- Engine angle
- Engine longitudinal position

$$L_I = R_I \delta$$

known in radians

$$d_o = \sin(\theta + \phi) L_2 + \frac{\sin^2(\theta + \phi)}{\cos(\theta + \phi)} L_1$$

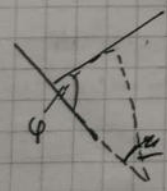
$$L_2 = L_{TB-N} + L_{Buffer} + k_2$$

? See prev. page.

$$L_1 = L_I + L_E + (r + r_{runner}) + L_R$$

fixed by 90° bend and draw

(12) Injector holder angle affects cp:



$$L_I = L_R + k_1$$

$$R_I (\pi - 2\phi) + L_R + k_1$$

(! easier to think about adjusting down cp)

$$\phi = \frac{\pi - \delta}{2}$$

(12) Protrusion in front of roll-trap plane:

$$d_o = \sin\left(\theta + \frac{\pi}{2} - \frac{\delta}{2}\right) (L_{TB-N} + L_{Buffer} + k_2) - \cos\left(\theta + \frac{\pi}{2} - \frac{\delta}{2}\right) (R_I \delta + L_R + k_1)$$

most easy to adjust - just line lengths