

Final Project Part 1

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4:22 PM

Takeoff:

$$P_{01} = 100 \text{ kPa}$$

$$T_{01} = 15^\circ \text{C} = 288 \text{ K}$$

$$\gamma = 1.4$$

$$C_p = 1004 \text{ J/kg K}$$

$$R = 287 \text{ J/kg K}$$

$$\text{compression ratio} = \frac{P_{02}}{P_{01}} = 6$$

$$\eta_{cp} = .92 \rightarrow \text{const}$$

$$\dot{m} = 20 \text{ kg/s}$$

Design Choices:

$$C_{b1} = 0$$

const axial velocity ($C_{x1} = C_{x2}$)

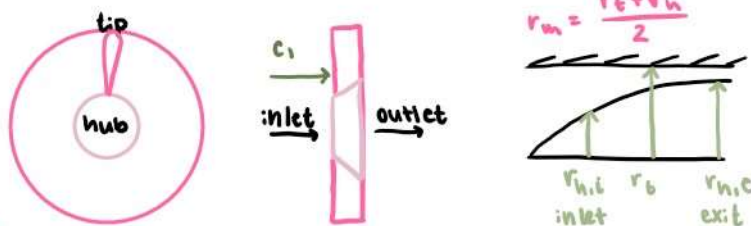
$$\left(\frac{r_{hub}}{r_{tip}}\right)_1 = K_1 = .6$$

$$M_{i1} = .34$$

$$U_{i1} = 259.2 \text{ m/s}$$

Size & Design 1st Stage of Compressor

1. size the compressor (D_b is const, D_h increases)



Inlet:

$$\frac{T_{01}}{T_i} = \left(1 + \frac{\gamma-1}{2} M_{i1}^2\right)^{-1}$$

$$T_i = T_{01} \left(1 + \frac{\gamma-1}{2} M_{i1}^2\right)^{-1}$$

$$T_i = 288 \text{ K} \left(1 + \frac{1.4-1}{2} (.34)^2\right)^{-1}$$

$$T_i = 281.49 \text{ K}$$

$$a_i = \sqrt{\gamma R T_i} = \sqrt{1.4 (287 \text{ J/kg K}) (281.49 \text{ K})} \quad a_i = 336.51 \text{ m/s}$$

$$M_{i1} = \frac{C_1}{a_i} \quad C_1 = M_{i1} a_i = 336.51 \text{ m/s} \cdot .34 \quad C_1 = 114.3 \text{ m/s}$$

$$\frac{P_{01}}{P_i} = \left(1 + \frac{\gamma-1}{2} M_{i1}^2\right)^{\frac{\gamma}{\gamma-1}}$$

$$P_i = P_{01} \left(1 + \frac{\gamma-1}{2} M_{i1}^2\right)^{-\frac{\gamma}{\gamma-1}}$$

$$P_i = 100 \text{ kPa} \left(1 + \frac{1.4-1}{2} (.34)^2\right)^{-1.4/1.4-1}$$

$$P_i = 92.31 \text{ kPa}$$

$$P_i = P_i R T_i \quad P_i = \frac{P_i}{R T_i}$$

$$P_i = \frac{92310 \text{ Pa}}{(287 \text{ J/kg K}) (281.49 \text{ K})} \quad P_i = 1.14 \text{ kg/m}^3$$

* $C_i = C_{x1}$ b/c $C_{\theta 1} = 0$

Inlet Annulus Area: $\dot{m} = P_i C_{i1} A_{i1}$

$$A_{i1} = \pi (r_{t1}^2 - r_{h1}^2)$$

$$A_{i1} = \pi r_{t1}^2 \left(1 - \frac{r_{h1}^2}{r_{t1}^2}\right)$$

$$r_{t1} = \sqrt{\frac{\dot{m}}{P_i C_{i1} \pi (1 - K_1^2)}} = \sqrt{\frac{20 \text{ kg/s}}{(1.14 \frac{\text{kg}}{\text{m}^3}) (114.3 \text{ m/s}) \pi (1 - .6^2)}}$$

$$r_{t1} = 0.28 \text{ m}$$

$$K = \frac{r_{h1}}{r_{t1}} \quad r_{h1} = K r_{t1} = (.6) (0.28 \text{ m}) \quad r_{h1} = 0.17 \text{ m}$$

Exit:

$$\frac{P_{02}}{P_{01}} = 6$$

$$P_{02} = 6 \cdot 100 \text{ kPa} = 600 \text{ kPa}$$

$$\eta_p = \frac{\gamma-1}{\gamma} \frac{\ln \left(\frac{P_{02}}{P_{01}}\right)}{\ln \left(\frac{T_{02}}{T_{01}}\right)}$$

$$e^{\ln \left(\frac{T_{02}}{T_{01}}\right)} = e^{\frac{\gamma-1}{\gamma \eta_p} \ln \left(\frac{P_{02}}{P_{01}}\right)}$$

$$T_{02} = T_{01} \left(\frac{P_{02}}{P_{01}}\right)^{\frac{\gamma-1}{\gamma \eta_p}}$$

$$\eta_p = \frac{\gamma-1}{\gamma} \frac{\ln(\frac{P_{0i}}{P_{0e}})}{\ln(\frac{T_{0e}}{T_{0i}})}$$

$$e^{\ln(\frac{T_{0e}}{T_{0i}})} = e^{\frac{\gamma-1}{\gamma} \eta_p \ln(\frac{P_{0i}}{P_{0e}})}$$

$$\frac{T_{0e}}{T_{0i}} = e^{(\frac{\gamma-1}{\gamma} \eta_p \ln(\frac{P_{0e}}{P_{0i}}))}$$

$$\frac{T_{0e}}{T_{0i}} = e^{\frac{1.4-1}{1.4 \times 0.92} \ln(6)} = 1.74$$

$$T_{0e} = 1.74(288K)$$

$$T_{0e} = 520K$$

$$\text{* } C_x \text{ is const } \therefore C_{xi} = C_{xe} = 114.3 \frac{m}{s}$$

$$C_{xe} = C_e$$

$$T_{0e} = T_e + \frac{C_e^2}{2C_p}$$

$$T_e = T_{0e} - \frac{C_e^2}{2C_p} = 520K - \frac{(114.3 m/s)^2}{2(1004 J/kgK)} \quad T_e = 495.9K$$

$$a_e = \sqrt{\gamma R T_e} = \sqrt{1.4(287 J/kgK)(495.9K)} \quad a_e = 446.37 m/s$$

$$M_e = \frac{C_e}{a_e} = \frac{114.3 m/s}{446.37 m/s} \quad M_e = 0.26$$

$$\frac{P_{0e}}{P_e} = (1 + \frac{\gamma-1}{2} M_e^2)^{\gamma/(\gamma-1)} \quad P_e = P_{0e} (1 + \frac{\gamma-1}{2} M_e^2)^{-\gamma/(\gamma-1)}$$

$$P_e = 600 kPa (1 + \frac{1.4-1}{2} (0.26)^2)^{-\frac{1.4}{1.4-1}}$$

$$P_e = 573.3 kPa$$

$$P_e = \frac{P}{R T_e} = \frac{573300}{(287 J/kgK)(495.9K)}$$

$$\rho_e = 4.03 kg/m^3$$

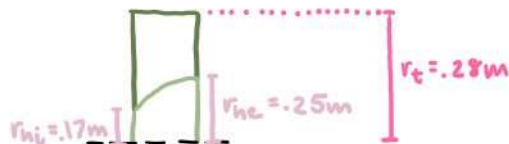
$$r_{0i} = r_{0e} = 0.28m$$

$$\dot{m} = \rho_e C_e A_e = \rho_e C_e \pi (r_{0e}^2 - r_{he}^2)$$

$$r_{he} = \sqrt{\frac{\dot{m}}{\rho_e C_e \pi} + r_{0e}^2}$$

$$r_{he} = \sqrt{\frac{-20 kg/s}{(4.03 kg/m^3)(114.3 m/s) \pi} + (0.28m)^2}$$

$$r_{he} = 0.25m \quad r_{0e} = 0.28m$$



2. Mean-line station

$$\frac{P_{03}}{P_{01}} = 1.1$$

$\dot{m} = 20 kg/s$	$U_0 = 259.2 m/s$
$T_{0i} = 288K$	$r_{hi} = 0.17m$
$T_i = 281.49K$	$r_{he} = 0.25m$
$a_i = 336.31 m/s$	$r_0 = 0.28m$
$M_i = 0.34$	$\gamma = 1.4$
$C_i = 114.3 m/s$	$R = 287 J/kgK$
$P_{0i} = 100 kPa$	$C_p = 1004 J/kgK$
$P_i = 92.31 kPa$	$K = 0.6$
$\rho_i = 1.14 kg/m^3$	$\eta_p = 0.92$
$T_{0e} = 520K$	
$T_e = 495.9K$	
$a_e = 446.37 m/s$	
$M_e = 0.26$	
$C_e = 114.3 m/s$	
$P_{0e} = 600 kPa$	
$P_e = 573.3 kPa$	
$\rho_e = 4.03 kg/m^3$	

$$P_c = 573.3 \text{ kPa}$$

$$\rho_c = 4.03 \text{ kg/m}^3$$

$$\cdot T_{03} = ?$$

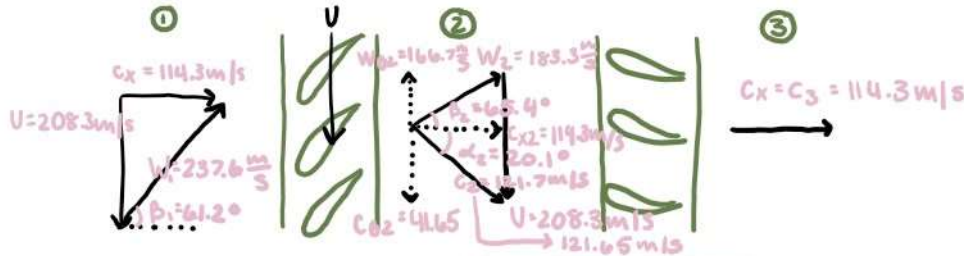
$$\eta_c = \frac{\left(\frac{P_{03}}{P_{01}}\right)^{\frac{\gamma-1}{\gamma}} - 1}{\frac{T_{03}}{T_{01}} - 1}$$

$$T_{03} = \left(\frac{\left(\frac{P_{03}}{P_{01}}\right)^{\frac{\gamma-1}{\gamma}} - 1}{\eta_c} + 1 \right) T_{01}$$

$$T_{03} = \left(\frac{(1.1)^{\frac{1.4-1}{1.4}} - 1}{.92} + 1 \right) (288 \text{ K})$$

$$T_{03} = 296.64$$

• Velocity Triangles:



$$w = \frac{U_b}{r_b} = \frac{259.2 \text{ m/s}}{.28 \text{ m}}$$

$$w = 925.7 \text{ rad/s}$$

$$r_m = \frac{(r_b + r_h)}{2} = \frac{(.28 \text{ m} + .17 \text{ m})}{2}$$

$$r_m = 0.225 \text{ m}$$

$$U_m = w r_m = (925.7 \text{ rad/s})(.225 \text{ m}) \quad U_m = 208.3 \text{ m/s}$$

$$\textcircled{1} \quad W_1 = \sqrt{U^2 + c_{x1}^2} = \sqrt{(208.3 \text{ m/s})^2 + (114.3 \text{ m/s})^2}$$

$$W_1 = 237.6 \text{ m/s}$$

$$\alpha_1 = 0^\circ$$

* $U_1 = U_2$ for axial flow machines

$$\beta_1 = \cos^{-1} \left(\frac{114.3 \text{ m/s}}{237.6 \text{ m/s}} \right)$$

$$\beta_1 = 61.2^\circ$$

②

$$\text{Euler Turbine: } \dot{W}_s = \dot{m} U (c_{\theta 2} - c_{\theta 1})$$

$$\text{Energy Eq: } \dot{W}_s = \dot{m} (h_{02} - h_{01}) = \dot{m} c_p (T_{02} - T_{01}) \quad T_{02} = T_{01} = 288 \text{ K}$$

$$\dot{m} U c_{\theta 2} = \dot{m} c_p (T_{03} - T_{01}) \quad c_{\theta 2} = \frac{c_p (T_{03} - T_{01})}{U}$$

$$c_{\theta 2} = \frac{1004 \text{ J/kgK} (296.6 \text{ K} - 288 \text{ K})}{208.3 \text{ m/s}}$$

$$c_{\theta 2} = 41.65 \text{ m/s}$$

$$c_2 = \sqrt{(41.65 \text{ m/s})^2 + (114.3 \text{ m/s})^2}$$

$$c_2 = 121.7 \text{ m/s}$$

$$\alpha_2 = \cos^{-1} \left(\frac{114.3 \text{ m/s}}{121.7 \text{ m/s}} \right) \quad \alpha_2 = 20.1^\circ$$

$$\dot{W}_s = \dot{m} c_p (T_{03} - T_{01}) = (20 \text{ kg/s})(1004 \text{ J/kgK})(296.6 \text{ K} - 288 \text{ K})$$

$$\dot{W}_s = 172688 \text{ J}$$

$$\tan \beta_2 = \frac{\dot{W}_s}{\dot{m} U c_x} + \tan \beta_1 \quad \beta_2 = \tan^{-1} \left(\frac{172688 \text{ J}}{(20 \frac{\text{kg}}{\text{s}})(208.3 \frac{\text{m}}{\text{s}})(114.3 \frac{\text{m}}{\text{s}})} + \tan(61.2^\circ) \right)$$

$$\beta_2 = 65.4^\circ$$

$$W_{02} = U - c_{\theta 2} = 208.3 \frac{\text{m}}{\text{s}} - 41.65 \frac{\text{m}}{\text{s}}$$

$$W_{02} = 166.7 \text{ m/s}$$

$$W_2 = \frac{W_{02}}{\sin \beta_2} = \frac{166.7 \text{ m/s}}{\sin(65.4^\circ)}$$

$$W_2 = 183.3 \text{ m/s}$$

• De-Haller Number:

$$w = 183.3 \text{ m/s}$$

$$D_r = 0.77 > .72$$

De-Haller Number:

Rotor $\rightarrow D_r = \frac{W_2}{W_1} = \frac{183.3 \text{ m/s}}{237.6 \text{ m/s}}$

$D_r = 0.77 > 0.72$
 \therefore the rotor design is good

Stator $\rightarrow D_s = \frac{C_2}{C_1} = \frac{114.3 \text{ m/s}}{121.7 \text{ m/s}}$

$D_s = 0.94 > 0.72 \therefore$ the stator design is good

Rotor Diffusion Factor:

$h/c = 3$
 $N_b = 30$

$D_r = 1 - \frac{W_2}{W_1} + \frac{|W_{02} - W_{01}|}{2\sigma_r W_1}$

$\sigma = \frac{c}{s} \quad s = \frac{2\pi r}{N}$

$h = r_b - r_{hi} = .28 \text{ m} - .17 \text{ m} \quad h = 0.11 \text{ m}$

$c = \frac{h}{s} = \frac{.11 \text{ m}}{3} \quad c = 0.037 \text{ m}$

$s = \frac{2\pi (.225 \text{ m})}{30} \quad s = 0.047 \text{ m}$

$D_r = 1 - .77 + \frac{|166.7 \text{ m/s} - 208.3 \text{ m/s}|}{2(.785)(237.6 \text{ m/s})}$

$D_r = .34 < .5 \therefore$ AR & blade # is good

$\sigma = \frac{.037 \text{ m}}{.047 \text{ m}} = .785$

Stator Diffusion Factor:

$h/c = 3$
 $N = 20$

$s = .07 \text{ m} \quad \sigma = \frac{.037 \text{ m}}{.07 \text{ m}} \quad \sigma = .52$

$s = \frac{2\pi (.225 \text{ m})}{20}$

$D_s = 1 - \frac{C_2}{C_1} + \frac{|C_{02} - C_{01}|}{2\sigma C_1}$

$D_s = 1 - .94 + \frac{0}{2(.52)(121.7 \text{ m/s})}$

$D_s = .06 < .5 \therefore$ AR & vane # is good

Stage Efficiency? Is $\eta_c = \eta_p$ a good assumption?

$\bar{w}_r = \bar{w}_s = .03$

$\bar{w}_r = \frac{(P_{02} - P_{01})_{rel}}{\frac{1}{2} \rho W_1^2}$

$\frac{T_{03}}{T_{01}} = \frac{296.64 \text{ K}}{288 \text{ K}} = \frac{T_{02}}{T_{01}} = 1.03$

$C_p T_{02} = C_p T_2 + \frac{C_2^2}{2}$

$\bar{w}_s = \frac{P_{03} - P_{01}}{\frac{1}{2} \rho C_2^2}$

$T_2 = T_{02} - \frac{C_2^2}{2C_p} = 296.64 \text{ K} - \frac{(21.7 \text{ m/s})^2}{2(1004 \text{ J/kg K})}$

$T_2 = 289.26 \text{ K}$

$a_2 = \sqrt{\gamma R T_2} = \sqrt{1.4(287 \text{ J/kg K})(289.26 \text{ K})}$

$a_2 = 340.9 \text{ m/s} \quad M_2 = \frac{C_2}{a_2} = \frac{121.7 \text{ m/s}}{340.9 \text{ m/s}} \quad M_2 = .36$

$T_1 = T_{01} - \frac{C_1^2}{2C_p} \quad M_{2,r} = \frac{W_2}{a_2} = \frac{183.3 \text{ m/s}}{340.9 \text{ m/s}} \quad M_{2,r} = .57$

$T_1 = 288 \text{ K} - \frac{(114.3 \text{ m/s})^2}{2(1004 \text{ J/kg K})} \quad T_1 = 281.5 \text{ m/s}$

$M_1 = \frac{C_1}{a_1} = \frac{114.3 \text{ m/s}}{336.3 \text{ m/s}} \quad M_1 = .34$

$M_{1,r} = \frac{W_1}{a_1} = \frac{237.6 \text{ m/s}}{336.3 \text{ m/s}}$

$a_1 = \sqrt{\gamma R T_1}$
 $a_1 = \sqrt{1.4(287 \text{ J/kg K})(281.5 \text{ m/s})}$
 $a_1 = 336.3 \text{ m/s}$

$$M_{1,r} = \frac{W_1}{a_1} = \frac{257.6 \text{ m/s}}{336.3 \text{ m/s}}$$

$$M_{1,r} = 0.71$$

$$\frac{P_{01}}{P_1} = \left(1 + \frac{\gamma-1}{2} M_{1,r}^2\right)^{\frac{\gamma}{\gamma-1}}$$

$$P_1 = P_{01} \left(1 + \frac{\gamma-1}{2} M_{1,r}^2\right)^{-\frac{\gamma}{\gamma-1}} = 100 \text{ kPa} \left(1 + \frac{1.4-1}{2} (.71)^2\right)^{-\frac{1.4}{1.4-1}}$$

$$P_1 = 92.3 \text{ kPa}$$

$$\rho_1 = \frac{P_1}{RT_1} = \frac{92300 \text{ Pa}}{(287 \frac{\text{J}}{\text{kgK}})(281.5 \frac{\text{m}}{\text{s}})} \quad \rho_1 = 1.14 \frac{\text{kg}}{\text{m}^3}$$

$$P_{01,r} = P_1 \left(1 + \frac{\gamma-1}{2} M_{1,r}^2\right)^{\frac{\gamma}{\gamma-1}}$$

$$P_{01,r} = 92.3 \text{ kPa} \left(1 + \frac{1.4-1}{2} (.71)^2\right)^{\frac{1.4}{1.4-1}}$$

$$P_{01,r} = 129.2 \text{ kPa}$$

$$\bar{w}_r = \frac{P_{01,r} - P_{02,r}}{\frac{1}{2}(\rho_1 W_1^2)}$$

$$-\bar{w}_r \cdot 5 \rho_1 W_1^2 + P_{01,r} = P_{02,r}$$

$$P_{02,r} = 129200 \text{ Pa} - .03(.5)(1.14 \frac{\text{kg}}{\text{m}^3})(257.6 \text{ m/s})^2$$

$$P_{02,r} = 129196 \text{ Pa}$$

$$P_2 = P_{02,r} \left(1 + \frac{\gamma-1}{2} M_{2,r}^2\right)^{-\frac{\gamma}{\gamma-1}}$$

$$P_2 = 129196 \text{ Pa} \left(1 + \frac{1.4-1}{2} (.57)^2\right)^{-\frac{1.4}{1.4-1}}$$

$$P_2 = 103646 \text{ Pa}$$

$$\frac{P_{02}}{P_2} = \left(1 + \frac{\gamma-1}{2} M_{2,r}^2\right)^{\frac{\gamma}{\gamma-1}}$$

$$P_{02} = 103646 \text{ Pa} \left(1 + \frac{1.4-1}{2} (.36)^2\right)^{\frac{1.4}{1.4-1}} \left(\frac{1.4}{1.4-1}\right)^{-1}$$

$$P_{02} = 113358 \text{ Pa}$$

$$\frac{P_{02}}{P_{01}} = \frac{113358 \text{ Pa}}{100,000 \text{ Pa}}$$

$$\frac{P_{02}}{P_{01}} = 1.13$$

$$\bar{w}_s = \frac{P_{02} - P_{03}}{.5 \rho_2 C_2^2}$$

$$\rho_2 = \frac{P_2}{RT_2} = \frac{103646 \text{ Pa}}{(287 \frac{\text{J}}{\text{kgK}})(289.2 \text{ K})}$$

$$\rho_2 = 1.25 \text{ kg/m}^3$$

$$P_{03} = P_{02} - \bar{w}_s \cdot 5 \rho_2 C_2^2$$

$$P_{03} = 113358 \text{ Pa} - .03(.5)(1.25 \frac{\text{kg}}{\text{m}^3})(121.7 \text{ m/s})^2$$

$$P_{03} = 113080 \text{ Pa}$$

$$\frac{P_{03}}{P_{02}} = \frac{113080 \text{ Pa}}{113358 \text{ Pa}}$$

$$\frac{P_{03}}{P_{02}} = 0.998$$

$$\frac{P_{03}}{P_{01}} = \left(\frac{P_{03}}{P_{02}}\right) \left(\frac{P_{02}}{P_{01}}\right) = (1.13)(.998)$$

$$\frac{P_{03}}{P_{01}} = 1.1$$

$$\frac{T_{03}}{T_{02}} = 1.03$$

$$\eta_c = \frac{\left(\frac{P_{03}}{P_{01}}\right)^{\frac{\gamma-1}{\gamma}} - 1}{\left(\frac{T_{03}}{T_{01}} - 1\right)} = \frac{(1.1)^{\frac{1.4-1}{1.4}} - 1}{1.03 - 1}$$

$\eta_c = 0.92 = \eta_p$ \therefore using polytropic efficiency was a good assumption

• find s'_2 & s'_3

efficiency was a good assumption

find δ_r^* & δ_s^*

$$\delta^* = m \theta \left(\frac{s}{c} \right)^n \quad n = .5 \quad \frac{a}{c} = .5 \quad \beta_2 = 65.4^\circ \quad \beta_1 = 61.2^\circ$$

$$m_r = .23 \left(\frac{2a}{c} \right)^2 + \frac{\beta_2}{500} \quad \alpha_2 = 0^\circ \quad m_r = .23(1)^2 + (65.4^\circ/500)$$

$$m_r = .26$$

$$m_s = .23 \left(\frac{2a}{c} \right)^2 + \frac{\alpha_3}{600} \quad m_s = .23(1)^2 + 0 = .23$$

$$c = 0 \rightarrow k_1 = \beta_1$$

$$\delta^* = \beta_2 - k_2$$

$$\theta = k_1 - k_2 = \beta_1 - k_2$$

$$\beta_1 - \beta_2 = \theta - \delta^* \quad \frac{s}{c} = \frac{1}{1.27} = 1.27$$

$$\beta_1 - \beta_2 = \theta - m \theta \left(\frac{s}{c} \right)^n$$

$$\frac{s}{c} = \frac{1}{.125} = 8$$

$$\beta_1 - \beta_2 = \theta (1 - m \left(\frac{s}{c} \right)^n)$$

$$\theta = \frac{\beta_1 - \beta_2}{1 - m \left(\frac{s}{c} \right)^n}$$

$$\theta_r = \frac{61.2^\circ - 65.4^\circ}{1 - .36(1.27)^.5} \cdot .5$$

$$\theta_r = -7.07^\circ$$

$$\alpha_2 = 20.1^\circ$$

$$\alpha_3 = 0^\circ$$

$$\theta_s = \frac{\alpha_3 - \alpha_2}{1 - m_s \left(\frac{s}{c} \right)^n} = \frac{-20.1^\circ}{1 - .23(8)^.5} \cdot .5 \quad \theta_s = -57.5^\circ$$

$$\delta_r^* = -\beta_1 + \beta_2 + \theta_r = 61.2^\circ + 65.4^\circ - 7.07^\circ \quad \delta_r^* = -2.87^\circ$$

$$\delta_s^* = -\alpha_2 + \alpha_3 + \theta_s = -20.1^\circ - 57.5^\circ \quad \delta_s^* = -77.6^\circ$$

Number of Stages: $\Delta T_{\text{stage}} = \text{const}$

$$T_c - T_c = 495.9K - 281.49K = 214.41K$$

$$T_3 - T_1 = 290K - 281.49K = 8.6K$$

$$T_3 = T_03 - \frac{C_3^2}{2C_p}$$

$$T_3 = 286.6K - \frac{114.3 \text{ m/s}^2}{2(1004 \text{ J/kgK})}$$

$$T_3 = 290K$$

$$\# \text{ Stages} = \frac{T_c - T_c}{T_3 - T_1} = \frac{214.41K}{8.6K}$$

$$\# \text{ Stages} = 24.9 \approx 25 \text{ Stages}$$

3. Create Velocity Triangles for tip & hub:

$$C_x = \text{const} = 114.3 \text{ m/s} \quad \omega = 925.7 \text{ rad/s}$$

Mean:

$$r_m C_\theta = .225m \cdot 41.65 \text{ m/s}$$

$$r C_\theta = 9.37$$

$$U_m = 208.3 \text{ m/s}$$

$$r C_\theta = \text{const}$$

$$\text{Tip: } r = .28m$$

$$C_\theta = 9.37 / .28$$

$$C_{\theta t} = 33.5 \text{ m/s}$$

$$U_t = \omega \cdot r_t = 925.7 \frac{\text{rad}}{\text{s}} (.28m)$$

$$U_t = 259.2 \text{ m/s}$$

$$\text{Hub: } r_h = .17m$$

$$C_\theta = 9.37 / .17m$$

$$C_{\theta h} = 55 \text{ m/s}$$

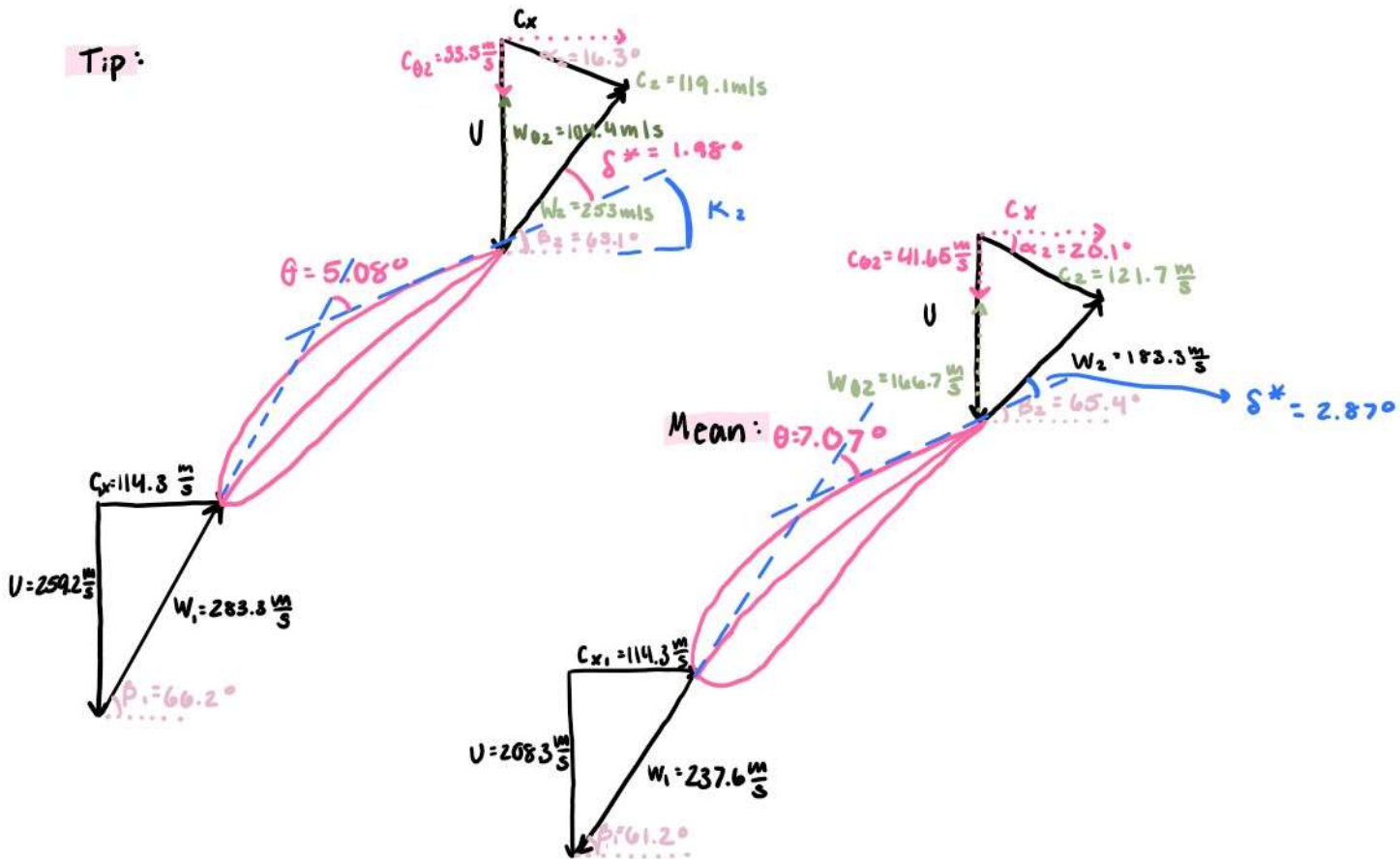
$$U_h = \omega r_h = 925.7 \frac{\text{rad}}{\text{s}} (.17m)$$

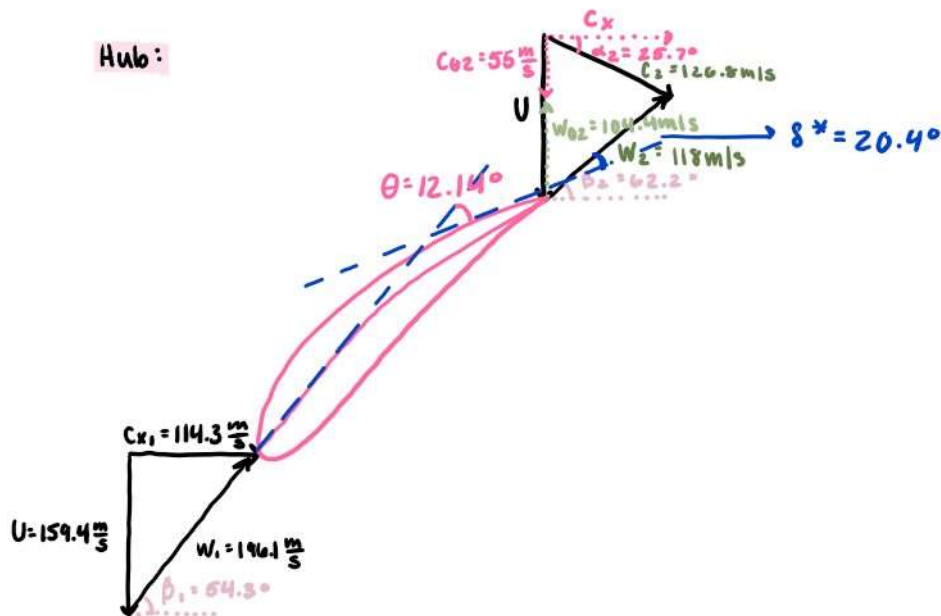
$$U_h = 159.4 \text{ m/s}$$

$$\begin{aligned}
 C_{0t} &= 33.5 \text{ m/s} \\
 U_b &= \omega \cdot r_b = 926.7 \frac{\text{rad}}{\text{s}} (1.28 \text{ m}) \\
 U_b &= 259.2 \text{ m/s} \\
 W_{02} &= U_b - C_{0t} = 259.2 \frac{\text{m}}{\text{s}} - 33.5 \frac{\text{m}}{\text{s}} \\
 W_{02} &= 225.7 \text{ m/s} \\
 W_2 &= \sqrt{225.7^2 + 114.3^2} \\
 W_2 &= 253 \text{ m/s} \\
 \beta_2 &= \sin^{-1}(225.7/253) \\
 \beta_2 &= 63.1^\circ \\
 C_2 &= \sqrt{33.5^2 + 114.3^2} \\
 C_2 &= 119.1 \text{ m/s} \\
 \alpha_2 &= \tan^{-1}(33.5/114.3) \\
 \alpha_2 &= 16.3^\circ \\
 W_1 &= \sqrt{259.2^2 + 114.3^2} \\
 W_1 &= 283.3 \text{ m/s} \\
 \beta_1 &= \cos^{-1}(114.3/283.3) \\
 \beta_1 &= 66.2^\circ
 \end{aligned}$$

$$\begin{aligned}
 C_{0n} &= 55 \text{ m/s} \\
 U_n &= \omega r_n = 926.7 \frac{\text{rad}}{\text{s}} (1.17 \text{ m}) \\
 U_n &= 159.4 \text{ m/s} \\
 W_{02} &= U_n - C_{0t} = 159.4 \frac{\text{m}}{\text{s}} - 55 \frac{\text{m}}{\text{s}} \\
 W_{02} &= 104.4 \text{ m/s} \\
 W_2 &= \sqrt{104.4^2 + 55^2} \\
 W_2 &= 118 \text{ m/s} \\
 \beta_2 &= \sin^{-1}(104.4/118) \\
 \beta_2 &= 62.2^\circ \\
 C_2 &= \sqrt{55^2 + 114.3^2} \\
 C_2 &= 126.8 \text{ m/s} \\
 \alpha_2 &= \tan^{-1}(55/114.3) \\
 \alpha_2 &= 25.7^\circ \\
 W_1 &= \sqrt{159.4^2 + 114.3^2} \\
 W_1 &= 196.1 \text{ m/s} \\
 \beta_1 &= \cos^{-1}(114.3/196.1) \\
 \beta_1 &= 54.3^\circ
 \end{aligned}$$

Tip:





4. Decide if rotor & stator are good designs

Tip:

$$D_{Rb} = \frac{W_{2b}}{W_{1b}} = \frac{253 \text{ m/s}}{283.3 \text{ m/s}}$$

$$D_{Rb} = 0.89 > 0.72$$

$$D_{sb} = \frac{C_{3b}}{C_{2b}} = \frac{114.3 \text{ m/s}}{119.1 \text{ m/s}}$$

$$D_{sb} = 0.96 > 0.72$$

Hub:

$$D_{Rh} = \frac{W_{2h}}{W_{1h}} = \frac{118 \text{ m/s}}{196.1 \text{ m/s}}$$

$$D_{Rh} = 0.6 < 0.72$$

$$D_{sh} = \frac{C_{3h}}{C_{2h}} = \frac{114.3 \text{ m/s}}{126.8 \text{ m/s}}$$

$$D_{sh} = 0.90 > 0.72$$

The tip rotor, tip stator & hub stator are good designs but the hub rotor is not

5. Deviation angle?

Tip:

$$\text{Rotor: } \delta^* = m\theta \left(\frac{s}{c}\right)^n$$

$$\frac{2\pi(1.28)}{30} = 1.58$$

$$\beta_1 - \beta_2 = \theta - \delta^*$$

$$66.2^\circ - 63.1^\circ = \theta - \left(0.23 + \frac{63.1^\circ}{500}\right)(1.58)$$

$$\frac{3.1^\circ}{1 - 0.39} = \theta = 5.08^\circ$$

$$\delta^* = -\beta_1 + \beta_2 + \theta$$

$$= -66.2^\circ + 63.1^\circ + 5.08^\circ$$

$$\delta_{Rb}^* = 1.98^\circ$$

Hub:

$$m = 0.23 \left(\frac{2.9}{c}\right)^2 + \frac{\beta_2}{500}$$

$$\beta_2 = 62.2^\circ$$

$$\beta_1 = 54.3^\circ$$

$$54.3^\circ - 62.2^\circ = \theta - \left(0.23 + \frac{62.2^\circ}{500}\right)(1.96)$$

$$\theta = -7.9^\circ$$

$$1 - 0.35$$

$$\delta^* = -\beta_1 + \beta_2 + \theta$$

$$= -54.3^\circ + 62.2^\circ - 7.9^\circ$$

$$\delta_{Rh}^* = -20.04^\circ$$

$$\text{Stator: } m = 0.23 \left(\frac{2.9}{c}\right)^2 + \frac{\beta_2}{500}$$

$$\frac{s}{c} = \frac{2\pi(1.28)}{30} = 0.24$$

$$\frac{s}{c} = \frac{2\pi(1.17)}{30} = 0.144$$

$$\text{Stator: } m = .23 \left(\frac{29}{c} \right)^2 + \frac{u_2}{600} \quad \frac{s}{c} = \frac{2\pi(1.28)}{20} = .24$$

$$m = .23$$

$$\alpha_2 - \alpha_3 = \theta - .23 \left(\frac{s}{c} \right)^n \theta$$

$$\frac{16.3^\circ}{1 - .23(.24)^5} = \theta \quad \theta = 18.38^\circ$$

$$\delta^* = -\alpha_2 + \theta = -16.3^\circ + 18.38^\circ$$

$$\delta_{st}^* = 2.1^\circ$$

$$\frac{s}{c} = \frac{2\pi(.17)}{20} = .144$$

$$\frac{25.7^\circ}{1 - .23(.144)^5} = \theta = 28.1^\circ$$

$$\delta^* = -\alpha_2 + \theta = -25.7 + 28.1^\circ$$

$$\delta_{sh}^* = 2.4^\circ$$

7. Comment on Velocity Triangles:

As you move from hub to mean to tip, the rotational velocity increases. Since the axial velocity (c_x) is constant, the relative velocity increases as the rotational velocity increases. Additionally, the camber decreases from hub to tip.