

# Complete Reference Guide: 4-Wheel Skid-Steer Rover Kinematics, Dynamics, Control & Turning Strategy

## 1 Geometry & 4WD Skid-Steer Kinematics

### 1.1 Notation

- $L$  = lateral distance between left & right wheel centers (track width)
- $T$  = longitudinal distance between front & rear axles (wheelbase length)
- Wheels indexed: LF, LR, RF, RR (left-front, left-rear, right-front, right-rear)
- $(v_L, v_R)$  = linear velocities of left-side and right-side virtual wheels (m/s) — usually desired commands
- $(x, y, \theta)$  = robot pose in world frame

### 1.2 Basic Body Kinematics

$$v = \frac{v_R + v_L}{2}, \quad \omega = \frac{v_R - v_L}{L} \quad (1)$$

Pose rates:

$$\dot{x} = v \cos \theta, \quad \dot{y} = v \sin \theta, \quad \dot{\theta} = \omega \quad (2)$$

### 1.3 Wheel Radii About ICC

Each physical wheel travels a different radius about the ICC (instantaneous center of curvature). If ICC is at distance  $R$  from robot center, wheel radii are:

$$R_{RF} = R - \frac{L}{2} - \frac{T}{2} \quad (3)$$

$$R_{RR} = R - \frac{L}{2} + \frac{T}{2} \quad (4)$$

$$R_{LF} = R + \frac{L}{2} - \frac{T}{2} \quad (5)$$

$$R_{LR} = R + \frac{L}{2} + \frac{T}{2} \quad (6)$$

Wheel linear speeds (geometric, no slip):

$$v_i = \omega R_i \quad \text{for wheel } i \quad (7)$$

In practice, lateral slip happens; real  $v_i$  from encoders will differ.

## 2 Skid (Lateral Slip) Modeling

When wheels are fixed (no steering), lateral motion creates tangential scrub forces. Two useful metrics:

## 2.1 Slip Ratio (Longitudinal)

For traction wheels on deformable ground — skip if hard floor.

## 2.2 Side-Slip/Skid Angle ( $\alpha$ )

For each wheel, the lateral force  $F_y$  grows with slip angle; approximate linear at small angles:

$$F_{y,i} \approx -C_{\alpha,i}\alpha_i \quad (8)$$

where  $C_\alpha$  is cornering stiffness (depends on tire and terrain).

**Practical implication:** Turning torque  $\gg$  ideal formula because you must overcome lateral friction at 4 contact patches.

## 3 Dynamics — Force/Torque Balance for 4WD

Let wheel radii =  $r$ . Motor torques  $T_{RF}, T_{RR}, T_{LF}, T_{LR}$ . Wheel forces at rim:

$$F_i = \frac{T_i}{r} \quad (9)$$

Total forward force:

$$F_x = \sum_i F_i - F_{\text{resist}} \quad (10)$$

Yaw moment:

$$M_z = \frac{L}{2} [(F_{RF} + F_{RR}) - (F_{LF} + F_{LR})] + \Delta_z \quad (11)$$

where  $\Delta_z$  accounts for asymmetries (CG offset etc.).

Equations of motion:

$$m\dot{v} = F_x, \quad I_z\dot{\omega} = M_z \quad (12)$$

## 4 Torque Required — Practical Formulas

### 4.1 Rolling Resistance

For baseline estimates on **flat ground**, to overcome rolling resistance and accelerate:

Rolling resistance force:

$$F_{\text{roll}} = \mu_r mg \quad (13)$$

Torque per wheel for steady speed & no turning:

$$T_{\text{wheel, steady}} = \frac{F_{\text{roll}}}{n_w} \cdot r \quad (14)$$

where  $n_w = 4$  wheels (assuming equal sharing).

### 4.2 Turning In Place

Turning in place ( $v_L = -v_R$ ) is worst-case scrub torque. Approximate extra torque per wheel from scrub:

Approximate *scrub friction torque* to rotate at low speed:

$$T_{\text{scrub, total}} \approx \mu_s mg \cdot \rho \quad (15)$$

where  $\rho$  is effective lever arm  $\sim$  half the diagonal footprint (rough approx).

A more conservative estimate: treat all four wheels dragging laterally so:

$$T_{\text{scrub, total}} \approx 4 \cdot (\mu_s N_i) \cdot r_{\text{eff}} \quad (16)$$

with  $N_i \approx \frac{mg}{4}$ ,  $r_{\text{eff}} \sim$  half the wheelbase or some small radius depending how scrub transmits to motor.

### 4.3 Practical Calculation Recipe

1. Measure/know:  $m$ ,  $r$ ,  $L$ ,  $T$ ,  $g$ ,  $\mu_r$  (rolling),  $\mu_s$  (static lateral), motor stall torque  $T_{\text{stall}}$ , gearbox ratio  $G$ , wheel radius  $r$ .
2. Required torque per wheel to move straight at steady speed (neglect accel):

$$T_{\text{req, straight}} \approx \frac{\mu_r mg}{n_w} \cdot r \quad (17)$$

3. For in-place rotation estimate (conservative):

$$T_{\text{req, turn}} \approx \frac{\mu_s mg}{2} \cdot \frac{L/2}{n_{\text{drive}}/2} \quad (18)$$

If  $T_{\text{stall}}$  (after gearing)  $< T_{\text{req}}$ , motors will stall or barely move.

## 5 Motor & Electrical Model

### 5.1 DC Motor Basics

$$T = K_t I \quad (19)$$

$$V = IR + K_e \omega_m \quad (20)$$

- Stall current:  $I_{\text{stall}} = \frac{V}{R}$  at zero speed
- Stall torque:  $T_{\text{stall}} = K_t I_{\text{stall}}$

## 6 Traction Control & Anti-Skid

- Monitor each wheel encoder. If one wheel's velocity  $\gg$  other wheel on same side and command equal, that wheel is slipping — reduce torque to that wheel (open-loop traction control).
- Use current sensing: sudden low current + high speed = slip.
- Use IMU yaw rate vs commanded  $\omega$  to detect slip and close the loop.

## 7 Example Numeric Walk-Through

Say:

- mass  $m = 15$  kg, wheel radius  $r = 0.05$  m,  $\mu_r = 0.03$ , 4 wheels

Compute rolling torque per wheel (steady):

$$F_{\text{roll}} = \mu_r mg = 0.03 \times 15 \times 9.81 \approx 4.41 \text{ N} \quad (21)$$

Torque per wheel:

$$T = \frac{F_{\text{roll}}}{4} \times r \approx \frac{4.41}{4} \times 0.05 \approx 0.055 \text{ N}\cdot\text{m} \quad (22)$$

(This is tiny — acceleration and turn will need far more.)

If you want to rotate in place and  $\mu_s \approx 0.6$  (rubber on concrete), very roughly:

$$\text{Yaw resisting moment} \approx \mu_s mg \cdot (L/2) \quad (23)$$

If  $L = 0.3$  m:

$$M_{\text{resist}} \approx 0.6 \times 15 \times 9.81 \times 0.15 \approx 13.2 \text{ N}\cdot\text{m} \quad (24)$$

Dividing by two driven sides then by two wheels per side gives torque per wheel  $\sim 13.2/4 \approx 3.3$  N·m. That's orders of magnitude larger than steady rolling torque above — shows why turning-in-place needs strong motors/gearbox.

## Part I

# Practical Turning Strategy for 50 kg Rover

## 8 Understanding the Situation

### 8.1 Key Constraints

- **In-place turn (yaw in place) is impossible:** Required torque per wheel  $\approx 22.3$  N·m, motor rated torque = 3.92 N·m.
- **Minimum turning radius along a curve** (forward + yaw)  $\approx 1.48$  m.
- Track width  $L = 0.52$  m, wheel radius  $r = 0.065$ – $0.07$  m.
- Motors are rated for 210 RPM output ( $\approx 1.54$  m/s linear at  $r = 0.07$  m).

**Strategy:** Turn while moving forward, not rotate in place.

## 9 Skid-Steer Kinematics for Turning

For a skid-steer vehicle:

$$v_L = v_c \left(1 - \frac{L}{2R}\right), \quad v_R = v_c \left(1 + \frac{L}{2R}\right) \quad (25)$$

Where:

- $v_L, v_R$  = linear speeds of left & right wheels
- $v_c$  = robot center forward speed
- $L$  = track width
- $R$  = turning radius

### 9.1 Example Calculation

For minimum radius  $R = 1.48$  m and track  $L = 0.52$  m:

Compute wheel speed ratio:

$$\frac{v_R}{v_L} = \frac{1 + L/(2R)}{1 - L/(2R)} = \frac{1 + 0.26/1.48}{1 - 0.26/1.48} \approx \frac{1 + 0.1757}{1 - 0.1757} = \frac{1.1757}{0.8243} \approx 1.426 \quad (26)$$

**Result:** The right wheels move  $\sim 1.43\times$  faster than left wheels to achieve that turn radius.

## 10 Speed Limitation Analysis

### 10.1 Torque Requirement

Max wheel torque must not exceed rated torque. Torque required decreases with forward motion because wheels are not fighting each other:

$$T_{\text{req,curve}} = T_{\text{req,in-place}} \cdot \frac{v_R - v_L}{v_R + v_L} \quad (27)$$

Substitute the ratio:

$$\frac{v_R - v_L}{v_R + v_L} = \frac{1.426 - 1}{1.426 + 1} = \frac{0.426}{2.426} \approx 0.175 \quad (28)$$

Torque required per wheel:

$$T_{\text{req,curve}} \approx 22.31 \cdot 0.175 \approx 3.90 \text{ N}\cdot\text{m} \quad (29)$$

**Result:** Matches rated torque perfectly. The rover can turn safely on this radius without overloading motors.

## 11 Implementation in Practice

### 11.1 Step 1: Set Motor Speeds Using Ratio

- Pick a forward center speed  $v_c \leq$  maximum allowable linear speed.
- Compute left & right wheel speeds:

$$v_L = \frac{2v_c}{1 + v_R/v_L}, \quad v_R = (v_R/v_L) \cdot v_L \quad (30)$$

- Use PWM motor control to match these wheel speeds.

### 11.2 Step 2: Smooth Acceleration

- Gradually ramp speeds to avoid spikes in torque.
- Start turn slowly  $\rightarrow$  avoid skidding or wheel slip.

### 11.3 Step 3: Verify with Low-Speed Test

- Use small trial  $v_c$  (e.g., 0.2 m/s)
- Observe turn radius
- Gradually increase until motors reach rated torque.

### 11.4 Step 4: Optional Improvements

- Slightly reduce radius by adding caster wheels (reduce lateral friction)
- Use trajectory planning: curve turns instead of abrupt point turns.

## 12 Summary

### 12.1 Key Takeaways

- **Cannot rotate in place** (torque too high)
- **Can turn on  $\sim 1.48$  m radius** by moving forward with wheel speed ratio  $\sim 1.426$
- **Forward motion reduces torque demand** from 22.31 N·m  $\rightarrow$  3.9 N·m per wheel
- **Safe with current motors** if you control speeds carefully

Parameter	Value/Conclusion
In-place rotation	Cannot rotate in place (torque too high)
Minimum turn radius	$\sim 1.48$ m
Wheel speed ratio	$\sim 1.426$ (right/left)
Torque reduction	From $22.31$ N·m $\rightarrow$ $3.9$ N·m per wheel
Safety	Safe with current motors if speeds controlled carefully

Table 1: Turning strategy summary for 50 kg rover