

1) Calculation of Pressurisation Flow-rate:

From R.F. Barron "Cryogenic Systems", translation from English language, Energoizdat, 1989.

Mass of gaseous Hydrogen required for Pressurisation to expel the liquid in the required time is  $\Delta mb$ .

$$\Delta mb = V_f * \rho_b^0 * ((T_o/T_{sat} - 1) * F1 * F2 + 1) * \exp(-F3),$$

where

$$V_f = 8.6 \text{ m}^3 - \text{the volume of displaced fluid}$$

$$T_o = 320 \text{ K} - \text{the inlet temperature of pressurization gas};$$

$$P_{\text{sat-subcool}} = \text{-- kPa-min pressure during subcooling}$$

$$T_{\text{sat}} = 15.0 \text{ K} - \text{the saturation temperature of Methane @ 30 kPa at initial pressure in the vessel};$$

$$P_{\text{pz}} = 2.20.E+07 \text{ Pa} - \text{the pressure of pressurization};$$

$$R = 4.1245 \text{ J/kg * K} - \text{the gas constant};$$

$$\rho_b^0 = 14.77208895 \text{ kg/m}^3 - \text{the density of pressurant gas @ } T_o \text{ and } P_{\text{pz}}$$

$$\gamma_{\text{pr}} = 1.416441341 - \text{specific heat ratio of pressurant hydrogen gas}$$

$$\gamma_{\text{liq}} = 1.241342342 - \text{specific heat ratio of methane liquid}$$

The functions F1, F2, F3 are determined as follows:

$$F1 = 1 - \exp(-0.330 * N1^{0.281});$$

$$F2 = 1 - \exp(-4.260 * N2^{0.857});$$

$$F3 = N3 * N2^{0.25} * ((1+N1) * (1+N2))^{0.25}.$$

The dimensionless factors N1, N2, N3 are determined as follows:

$$N1 = \rho_w * c_w^0 * t_w * T_{\text{sat}} / (\rho_d^0 * c_d^0 * D * T_o);$$

$$N2 = hc * \Theta * T_{\text{sat}} / (\rho_d^0 * c_d^0 * D * T_o);$$

$$N3 = Q * \Theta / (\rho_d^0 * c_d^0 * D * T_o);$$

where

$$\rho_w = 7800 \text{ kg/m}^3 - \text{the density of vessel material};$$

$$c_w^0 = 465 \text{ J/kg * K} - \text{the heat capacity of vessel material at } T_o;$$

$$t_w = 0.11 \text{ m} - \text{the thickness of vessel wall};$$

$$c_d^0 = 14745.63 \text{ J/kg * K} - \text{the heat capacity of pressurization gas at } T_o;$$

$$\rho_d^0 = 14.77 \text{ kg/m}^3 - \text{the density of pressurization gas};$$

$$D = 1.6 \text{ m} - \text{the inner diameter of vessel};$$

$$\Theta = 50 \text{ s} - \text{the time of liquid outflow};$$

$$L = 2.35 \text{ m} - \text{the average height of vessel wall, which is flowed around by gas}.$$

The average heat – transfer factor may be determined from the conditions of turbulent free convection near the vertical surface:

$$Nu = hc * L / \lambda = 0.13 * (Gr * Pr)^{1/3}.$$

$$\text{The behavior of gas at } T_{\text{avg}} = \frac{(T_o + T_{\text{sat}})}{2} = 167.5 \text{ K and } P_{\text{pz}} = 2.20.E+07 \text{ Pa}$$

$$\rho = 26.47629809 \text{ kg/m}^3 - \text{the density of gas};$$

$$\lambda = 1.44E-01 \text{ W/m*K} - \text{the thermal conductivity of gas};$$

$$\mu = 7.10E-06 \text{ N*s/m}^2 - \text{the gas dynamic viscosity};$$

$$Pr = 0.70 - \text{the Prandtl criterion};$$

$$\nu = 2.68E-07 \text{ m}^2/\text{s} - \text{the gas kinematic viscosity};$$

$$\beta = 5.97E-03 \text{ Coefficient of volume expansion equals } (1/T) \text{ for ideal gases}$$

The criterion Gr is the following:

$$Gr = \beta * L^3 * g * (T_o - T_{\text{sat}}) / \nu^2 = 3.22612E+15$$

$$Nu = 1.71E+04$$

$$hc = Nu * \lambda / L = 1044.46 \text{ W/m}^2\text{K} - \text{the convective heat transfer between the vessel wall and gas};$$

$$Q = hc * (T_o - T_{\text{sat}}) = 3.19E+05 \text{ W/m}^2 - \text{the heat transfer related to surface area unit};$$

$$N1 = 0.053660692 \quad N1 = \rho_w * c_w^0 * t_w * T_{\text{sat}} / (\rho_d^0 * c_d^0 * D * T_o);$$

$$N2 = 0.007023921 \quad N2 = hc * \Theta * T_{\text{sat}} / (\rho_d^0 * c_d^0 * D * T_o);$$

$$N3 = 0.142819732 \quad N3 = Q * \Theta / (\rho_d^0 * c_d^0 * D * T_o);$$

$$F1 = 0.135029514 \quad F1 = 1 - \exp(-0.330 * N1^{0.281});$$

$$F2 = 0.058991839 \quad F2 = 1 - \exp(-4.260 * N2^{0.857});$$

$$F3 = 0.041963168 \quad F3 = N3 * N2^{0.25} * ((1+N1) * (1+N2))^{0.25}.$$

$$\Delta mb = 141.5500938 \text{ kg} \quad \Delta mb = V_f * \rho_b^0 * ((T_o/T_{\text{sat}} - 1) * F1 * F2 + 1) * \exp(-F3),$$

$$\text{The average flow rate of pressurization is } G_{\text{pz}} = \Delta mb / \Theta = 2.83 \text{ kg/s}$$

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