

US012390909B2

(12) United States Patent Hirabayashi et al.

(54) IMPACT TOOL

(71) Applicant: MAKITA CORPORATION, Anjo (JP)

(72) Inventors: Tokuo Hirabayashi, Anjo (JP); Haruki Tejima, Anjo (JP); Kazuma Aoki, Anjo

(JP); **Kazunori Kinoshita**, Anjo (JP)

(73) Assignee: MAKITA CORPORATION, Anjo (JP)

(*) Notice: Subject to any disclaimer, the term of this

patent is extended or adjusted under 35 LLS C 154(b) by 9 days

U.S.C. 154(b) by 9 days.

This patent is subject to a terminal dis-

claimer.

(21) Appl. No.: 18/599,282

(22) Filed: Mar. 8, 2024

(65) **Prior Publication Data**

US 2024/0208013 A1 Jun. 27, 2024

Related U.S. Application Data

(63) Continuation of application No. 18/060,094, filed on Nov. 30, 2022, now Pat. No. 11,958,170.

(30) Foreign Application Priority Data

Dec. 13,	2021	(JP)		2021-201914
			•••••	

(51) Int. Cl. B25F 5/00 (2006.01) B25B 21/02 (2006.01)

(Continued)

(10) Patent No.: US 12,390,909 B2

(45) **Date of Patent:** *Aug. 19, 2025

(58) Field of Classification Search

CPC B25B 21/02 See application file for complete search history.

(56) References Cited

U.S. PATENT DOCUMENTS

8,371,848 B2 2/2013 Okawa et al. 8,496,366 B2 7/2013 Leong (Continued)

FOREIGN PATENT DOCUMENTS

DE 202015007649 U1 * 12/2015 B25B 21/02 JP 2004174635 A 6/2004 (Continued)

OTHER PUBLICATIONS

Office Action mailed Aug. 30, 2023, in related U.S. Appl. No. 18/060,094, and examined claims 1-18.

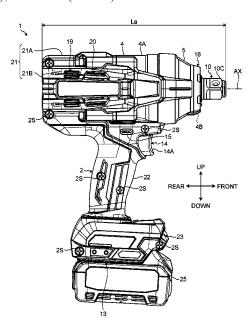
(Continued)

Primary Examiner — Eyamindae C Jallow (74) Attorney, Agent, or Firm — J-TEK LAW PLLC; Jeffrey D. Tekanic; Scott T. Wakeman

(57) ABSTRACT

An impact tool includes: a brushless motor; a spindle, which is rotated by the brushless motor; a hammer, which is held on the spindle; an anvil, which is impacted in a rotational direction by the hammer, a polymer housing, which houses the brushless motor; a hammer case, which is connected to the polymer housing and houses the hammer and the spindle; and a light unit, which is held on the hammer case and comprises a plurality of light-emitting devices. The maximum tightening torque of the anvil is 1,000 N·m or more and 2,500 N·m or less.

22 Claims, 9 Drawing Sheets



US 12,390,909 B2 Page 2

(51)	Int. Cl. B25B 23/18 B25F 5/02	3	(2006.01) (2006.01)		2021/00941 2021/01220 2021/02228 2021/03797 2022/00090	17 A1 66 A1 43 A1	4/2021 7/2021	Niwa Schneider et al.
(56)		Referen	ices Cited		2022/00408	29 A1	2/2022	Schneider et al.
	II C	DATENIT	DOCUMENTS		2022/01906/ 2022/01938			Fukuoka Tomayko et al.
	0.5	. PALENT	DOCUMENTS		2022/02035		6/2022	Kondo et al.
g	0,028,088 B2	5/2015	Vanko et al.		2022/03881	34 A1	12/2022	
),774,273 B2		Suzuki et al.		2023/01475	98 A1	5/2023	Tamura et al.
	0,525,582 B2		Qiu et al.		2023/01822	69 A1	6/2023	Hirabayashi et al.
	,453,106 B2	9/2022			2023/01822	70 A1		Hirabayashi et al.
	0002934 A1		Hsu	D25D 21/00	2023/02565			Yang et al.
2003/	0002934 AT	1/2003	11su	408/124				
2003/	/0058638 A1	* 3/2003	Gillette	B25B 23/18	F	FOREIG	n patei	NT DOCUMENTS
				362/202				
2003/	/0095402 A1	* 5/2003	Lin	B25B 23/18	JP	2014107	965 A	6/2014
				362/253	JP	2015066	636 A	4/2015
2004/	0174699 A1	* 9/2004	Minalga	B25B 23/18	JP	2015139	827 A	8/2015
			C	362/119	JP	5900	141 B2	4/2016
2005	/0157489 A1	* 7/2005	Oomori		JP	2016097	498 A	5/2016
2005	0137 105 711	112003	Comon	362/119	JP	201818	374 A	11/2018
2008	/0266845 A.1	* 10/2008	Wu		JP	2021024	044 A	2/2021
2000	02000 1 3 A1	10/2008	νν u	362/157	JP	2021112	816 A	8/2021
2010	/0140700 A1	sk 6/2010	T		WO	2011013	853 A2	2/2011
2010/	/0149790 A1	* 6/2010	Leong		WO	2023086		5/2023
2010	/0224033 A1	0/2010	Yeoh et al.	362/119	0	2025000	257 111	5,2025
	/0224033 A1 /0180286 A1		Oomori et al.			OTI	IPD DIE	DI ICATIONE
						OH	IEK PU	BLICATIONS
	/0187211 A1		Matsunaga et al.					
	/0188232 A1		Friedman et al.					23, in U.S. Appl. No. 18/060,103,
	/0036482 A1		Vanko et al.		and examined			
	0182869 A1	7/2014	Kumagai et al.		Office Action	mailed	Mar. 14,	2024, in related U.S. Appl. No.
	/0069864 A1		Nagahama et al.		18/060,103, a			
2015	/0283685 A1	* 10/2015	Kynast					e Patent Office dispatched May 7,
				362/119				pplication No. 2021-201915, and
2016	/0204718 A1	7/2016	Koizumi et al.		machine trans			pprication 110. 2021-201919, and
2016	/0354889 A1	12/2016	Ely et al.					D + + Off - 11 + 13 f - 12
2017/	/0173768 A1	* 6/2017	Dey, IV	B25B 21/02				Patent Office dispatched May 13,
2018/	/0126538 A1		Qiu et al.				apanese a	application No. 2021-201914, and
	/0222022 A1		Kumagai et al.		translation the	ereof.		
	/0091824 A1		Chellew					
	/0060755 A1		Kawai		* cited by e	xaminer		

^{*} cited by examiner

FIG.1

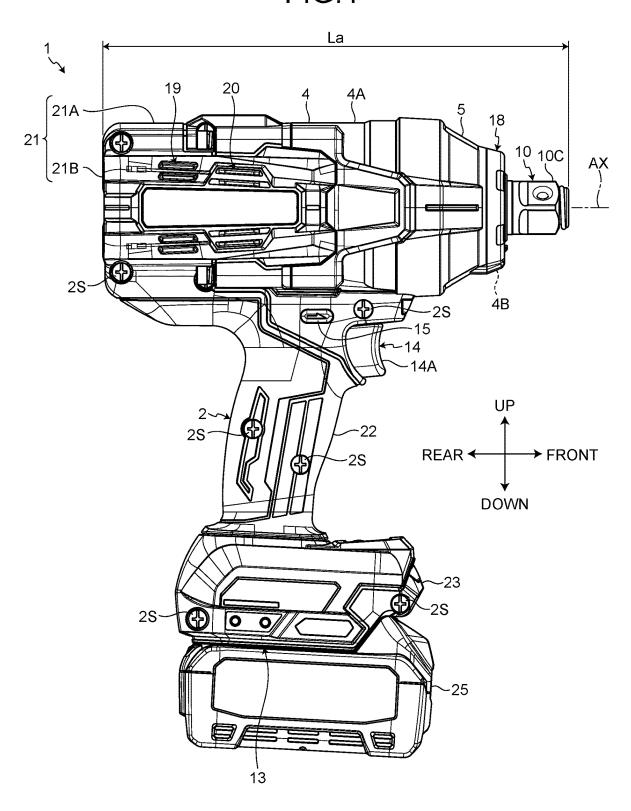


FIG.2

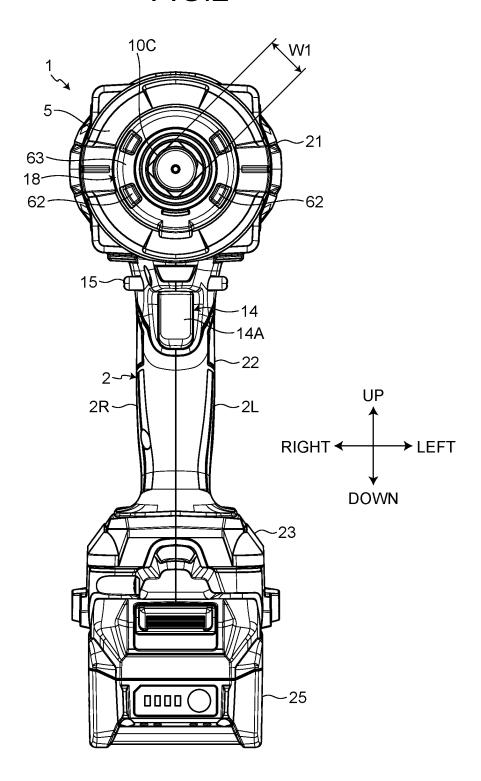
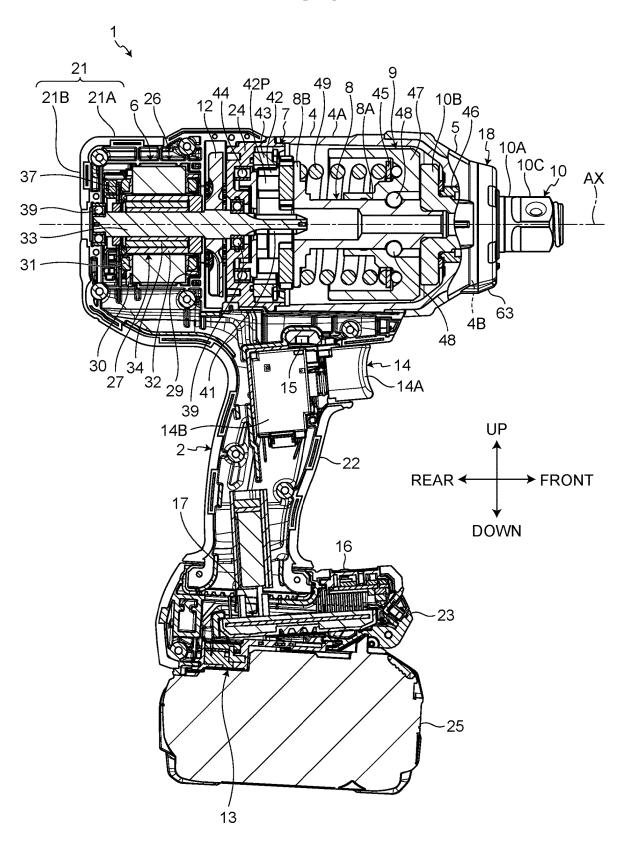
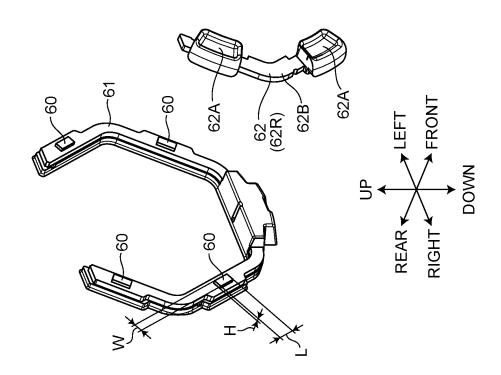


FIG.3



62A (62L) (62L) 63A 63A 63A 63A

FIG.4



Aug. 19, 2025				Sheet 5 of 9						US 12,39		
Wrench Width [inch]	0.25	0.5	0.5	_	_	0.375	0.5	0.375	0.5	0.375	0.5	0.5
Maximum Rotational Speed [rpm]	2800	2850	1900	1200	1650	2400	1900	2600	2800	2700	1700	1900
Overall Length [mm]	129.5	133.4	223.8	455.7	276.9	121.9	254	139.7	170.18	150	166	238
Weight [kg]	0.95	0.91	2.72	9.52	4.76	1.09	2.36	0.73	1.09	0.8	12	2.4
Rated Voltage [V]	18	12	18	18	18	18	18	12	18	12	18	18
Maximum Tightening Torque [Nm]	192	339	949	2576	2034	339	8/9	115	181	100	260	1057
Number of Light- Emitting Devices	3	3	l	3	1	3	l	3	1	l	1	0
Product	А	В	၁	Q	Ш	Ш	9	エ	_	ſ	Х	٦
Company	۵		В			>			Ю			

FIG.6

[Relationship Between Maximum Tightening Torque and Number of Light-Emitting Devices]

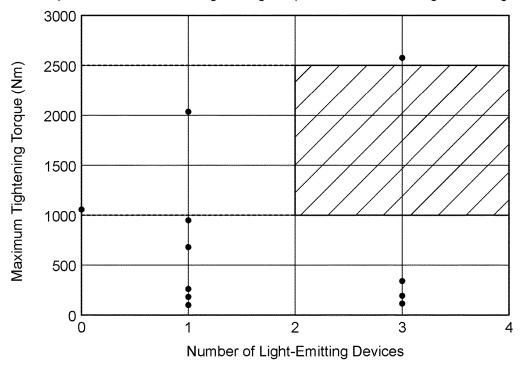


FIG.7

[Relationship Between Weight and Number of Light-Emitting Devices]

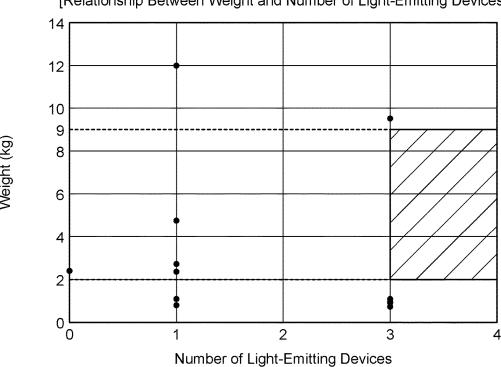


FIG.8

Aug. 19, 2025

[Relationship Between Overall Length and Number of Light-Emitting Devices]

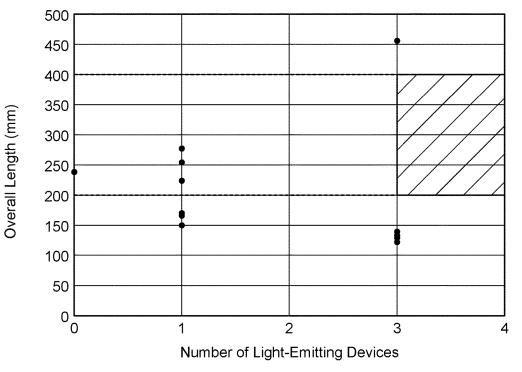


FIG.9

[Relationship Between Maximum Rotational Speed and Number of Light-Emitting Devices]

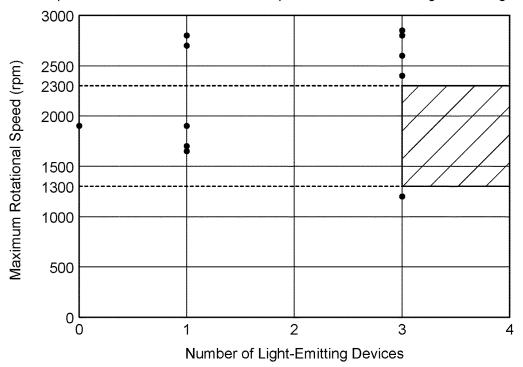


FIG.10

Aug. 19, 2025

[Relationship Between Wrench Width and Number of Light-Emitting Devices]

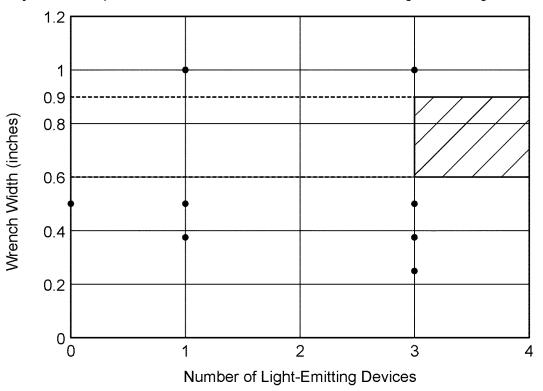


FIG.11

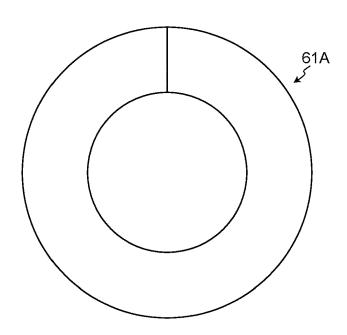


FIG.12

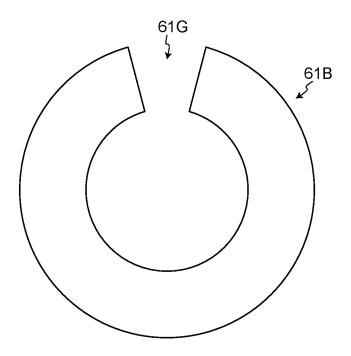
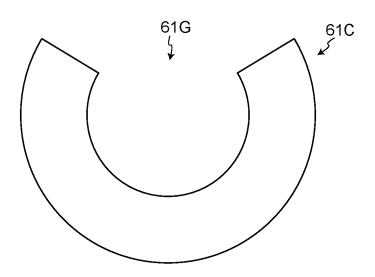


FIG.13



1 IMPACT TOOL

CROSS-REFERENCE

This application is a continuation of U.S. patent application Ser. No. 18/060,094, filed on Nov. 30, 2022, now U.S. Pat. No. 11,958,170, which claims priority to Japanese Patent Application No. 2021-201914 filed on Dec. 13, 2021, and to Japanese Patent Application No. 2021-201915 filed on Dec. 13, 2021, the contents of both of which are incorporated herein by reference.

TECHNICAL FIELD

Techniques disclosed in the present specification relate to an impact tool.

BACKGROUND ART

A known impact driver comprising lights is disclosed in Japanese Patent No. 5900141.

SUMMARY

It is one non-limiting object of the present teachings to disclose techniques for improving the ergonomics and/or work efficiency of an impact tool, such as an impact wrench and/or an impact driver.

In one non-limiting aspect of the present teachings, an 30 impact tool, such as an impact wrench or impact driver, may comprise: a brushless motor; a spindle, which is rotated by the brushless motor; a hammer, which is held on (and/or around) the spindle; an anvil, which is (configured to be) impacted in a rotational direction by the hammer; a resin 35 (polymer) housing, which houses the brushless motor; and a hammer case, which is connected to the resin (polymer) housing and houses the hammer and the spindle. The impact tool may comprise a light unit, which is held on the hammer case and comprises a plurality of light-emitting devices. The 40 maximum tightening torque of the anvil may be 1,000 N·m or more and 2,500 N·m or less.

According to the techniques disclosed in the present specification, an impact tool having improved ergonomics and/or work efficiency is provided.

BRIEF DESCRIPTION OF THE DRAWINGS

- FIG. 1 is a side view that shows an impact tool according to one representative, non-limiting embodiment of the present teachings.
- FIG. 2 is a front view that shows the impact tool according to the embodiment.
- FIG. 3 is a cross-sectional view that shows the impact tool according to the embodiment.
- FIG. 4 is an exploded, oblique view, viewed from the front, that shows a light unit according to the embodiment.
- FIG. 5 is a table that shows the specifications of a variety of known impact tools.
- FIG. **6** is a graph that shows the relationship between 60 maximum tightening torque of an anvil and number of light-emitting devices according to the above-described known impact tools and the embodiment.
- FIG. 7 is a graph that shows the relationship between weight of the impact tool and number of light-emitting 65 devices according to the above-described known impact tools and the embodiment.

2

- FIG. 8 is a graph that shows the relationship between overall length of the impact tool and number of light-emitting devices according to the above-described known impact tools and the embodiment.
- FIG. 9 is a graph that shows the relationship between maximum rotational speed of the anvil and number of light-emitting devices according to the above-described known impact tools and the embodiment.
- FIG. 10 is a graph that shows the relationship between wrench width of the anvil and number of light-emitting devices according to the above-described known impact tools and the embodiment.
- FIG. 11 is a drawing that schematically shows a modified example of a light circuit board
- FIG. 12 is a drawing that schematically shows another modified example of the light circuit board.
- FIG. 13 is a drawing that schematically shows another modified example of the light circuit board.

DETAILED DESCRIPTION

As was mentioned above, in one or more embodiments, an impact tool may comprise: a brushless motor; a spindle, which is rotated by the brushless motor; a hammer, which is held on the spindle; an anvil, which is impacted in a rotational direction by the hammer; a resin (polymer) housing, which houses the brushless motor; and a hammer case, which is connected to the resin (polymer) housing and houses the hammer and the spindle. The impact tool may comprise a light unit, which is held on the hammer case and comprises a plurality of light-emitting devices. The maximum tightening torque of the anvil may be 1,000 N·m or more and 2,500 N·m or less.

According to the above-mentioned configuration, because the light unit comprises the plurality of light-emitting devices, the work environment is brightly illuminated with illumination light. In addition, the maximum tightening torque of the anvil is 1,000 N·m or more and 2,500 N·m or less. Consequently, an impact tool having improved ergonomics and/or work efficiency is provided.

In one or more embodiments, at least three of the lightemitting devices may be provided.

According to the above-mentioned configuration, because the light unit comprises at least three of the light-emitting devices, the work environment is brightly illuminated with illumination light. Consequently, an impact tool having improved ergonomics and/or work efficiency is provided.

In one or more embodiments, a battery pack may be mounted on a battery-holding part, which is a portion of the resin (polymer) housing. The weight of the impact tool may be 2 kg or less.

According to the above-mentioned configuration, because the impact tool, with the battery pack mounted thereon, has a weight of 2 kg or less, an impact tool having improved ergonomics and/or work efficiency is provided.

In one or more embodiments, a battery pack may be mounted on a battery-holding part, which is a portion of the resin (polymer) housing. At least three of the light-emitting devices may be provided. The weight of the impact tool may be 2 kg or more and 9 kg or less.

According to the above-mentioned configuration, because the light unit comprises at least three of the light-emitting devices and the impact tool, with the battery pack mounted thereon, has a weight of 2 kg or more and 9 kg or less, an impact tool having improved ergonomics and/or work efficiency is provided.

In one or more embodiments, the spindle may be disposed forward of the brushless motor; the anvil may be disposed forward of the spindle; and the brushless motor may be housed in a motor-housing part, which is a portion of the resin (polymer) housing. The distance from a front-end 5 portion of the anvil to a rear-end portion of the motor-housing part may be 155 mm or less.

According to the above-mentioned configuration, because the overall length of the impact tool, which is defined as the distance between the front-end portion of the anvil and the 10 rear-end portion of the motor-housing part, is 155 mm or less, an impact tool having improved ergonomics and/or work efficiency is provided.

In one or more embodiments, the spindle may be disposed forward of the brushless motor; the anvil may be disposed 15 forward of the spindle; and the brushless motor may be housed in a motor-housing part, which is a portion of the resin (polymer) housing. At least three of the light-emitting devices may be provided. The distance from a front-end portion of the anvil to a rear-end portion of the motor-housing part may be 200 mm or more and 400 mm or less.

According to the above-mentioned configuration, because the light unit comprises at least three of the light-emitting devices and the overall length of the impact tool, which is defined as the distance from the front-end portion of the 25 anvil to the rear-end portion of the motor-housing part, is 200 mm or more and 400 mm or less, an impact tool having improved ergonomics and/or work efficiency is provided.

In one or more embodiments, the maximum rotational speed of the anvil may be 3,000 rpm or less.

According to the above-mentioned configuration, because the maximum rotational speed of the anvil is 3,000 rpm or less, an impact tool having improved ergonomics and/or work efficiency is provided.

In one or more embodiments, at least three of the lightemitting devices may be provided. The maximum rotational speed of the anvil may be 1,300 rpm or more and 2,300 rpm or less.

According to the above-mentioned configuration, because the light unit comprises at least three of the light-emitting 40 devices and the maximum rotational speed of the anvil is 1,300 rpm or more and 2,300 rpm or less, an impact tool having improved ergonomics and/or work efficiency is provided.

In one or more embodiments, the anvil may have a 45 quadrangular-column part. The dimension of one side of a cross section of the quadrangular-column part may be 0.375 inches or less.

According to the above-mentioned configuration, because the dimension of one side of a cross section of the quadrangular-column part is 0.375 inches or less, an impact tool having improved ergonomics and/or work efficiency is provided

In one or more embodiments, the anvil may have a quadrangular-column part. At least three of the light-emit- 55 ting devices may be provided. The dimension of one side of a cross section of the quadrangular-column part may be 0.6 inches or more and 0.9 inches or less.

According to the above-mentioned configuration, because the light unit comprises at least three of the light-emitting 60 devices and the dimension of one side of a cross section of the quadrangular-column part is 0.6 inches or more and 0.9 inches or less, an impact tool having improved ergonomics and/or work efficiency is provided.

In one or more embodiments, the light unit may comprise: 65 a light circuit board, which is disposed at least partially around the hammer case and holds the plurality of light-

4

emitting devices; and optical members, which are disposed forward of the light-emitting devices and the light circuit board

According to the above-mentioned configuration, the light-emitting devices and the light circuit board are protected by the optical members.

An embodiment is explained below, with reference to the drawings. In the embodiment, positional relationships among the parts are explained using the terms left, right, front, rear, up, and down. These terms indicate relative position or direction, wherein the center of an impact tool 1 is the reference. The impact tool 1 comprises a motor 6, which serves as a motive power supply.

In the embodiment, the direction parallel to rotational axis AX of the motor 6 is called the axial direction where appropriate, the direction that goes around rotational axis AX is called the circumferential direction or the rotational direction where appropriate, and the radial direction of rotational axis AX is called the radial direction where appropriate.

Rotational axis AX extends in a front-rear direction. One side in the axial direction is forward, and the other side in the axial direction is rearward. In addition, in the radial direction, a location that is proximate to or a direction that approaches rotational axis AX is called radially inward where appropriate, and a location that is distant from or a direction that leads away from rotational axis AX is called radially outward where appropriate.

Impact Tool

FIG. 1 is a side view that shows the impact tool 1 according to one representative, non-limiting embodiment of the present teachings. FIG. 2 is a front view that shows the impact tool 1 according to the embodiment. FIG. 3 is a cross-sectional view that shows the impact tool 1 according to the embodiment. In the embodiment, the impact tool 1 is an impact wrench.

The impact tool 1 comprises a housing 2, a hammer case 4, a hammer-case cover 5, the motor 6, a speed-reducing mechanism 7, a spindle 8, an impact mechanism 9, an anvil 10, a fan 12, a battery-mounting part 13, a trigger switch 14, a forward/reverse-change lever 15, an action-mode-change switch 16, a controller 17, and a light unit 18.

The housing 2 houses at least the motor 6. The housing 2 is made of a synthetic resin (polymer), such as nylon. Thus, the housing 2 is a resin (polymer) housing. The housing 2 comprises a pair of half housings. The housing 2 comprises a left housing 2L and a right housing 2R, which is disposed rightward of the left housing 2L. The left housing 2L and the right housing 2R are fixed to each other by a plurality of screws 2S.

The housing 2 comprises a motor-housing part 21, a grip part 22, and a battery-holding part 23.

The motor-housing part 21 houses the motor 6. The motor-housing part 21 comprises: a tubular part 21A, which is disposed around the motor 6; and a rear-cover part 21B, which is disposed at a rear-end portion of the tubular part 21A.

The grip part 22 is gripped by a user. The grip part 22 protrudes downward from the tubular part 21A. The trigger switch 14 is provided at an upper portion of the grip part 22.

The battery-holding part 23 holds a battery pack 25 via the battery-mounting part 13. The battery-holding part 23 houses the controller 17. The battery-holding part 23 is connected to a lower-end portion of the grip part 22. In both the front-rear direction and the left-right direction, the

dimension of the outer shape of the battery-holding part 23 is larger than the dimension of the outer shape of the grip part 22.

The hammer case 4 houses the speed-reducing mechanism 7, the spindle 8, the impact mechanism 9, and at least 5 a portion of the anvil 10. The hammer case 4 is made of metal. The hammer case 4 has a tube shape. The hammer case 4 is connected to a front portion of the motor-housing part 21. The hammer case 4 is sandwiched between the left housing 2L and the right housing 2R. A bearing box 24 is 10 fixed to a rear portion of the hammer case 4. The bearing box 24 is fixed to both the motor-housing part 21 and the hammer case 4.

The hammer-case cover 5 covers at least a portion of the surface of the hammer case 4. The hammer-case cover 5 is 15 made of a synthetic resin (polymer), such as nylon. The hammer-case cover 5 protects the hammer case 4. The hammer-case cover 5 blocks contact between the hammer case 4 and objects around the impact tool 1. The hammer-case cover 5 blocks contact between the hammer case 4 and 20 the user.

The motor **6** is the motive power supply of the impact tool **1**. The motor **6** is an inner-rotor-type brushless motor. The motor **6** is housed in the motor-housing part **21**, which is a portion of the housing **2**.

The motor 6 comprises a stator 26 and a rotor 27. The stator 26 is supported in the motor-housing part 21. At least a portion of the rotor 27 is disposed in the interior of the stator 26. The rotor 27 rotates relative to the stator 26. The rotor 27 rotates about rotational axis AX, which extends in 30 the front-rear direction.

The stator 26 comprises a stator core 28, a front insulator 29, a rear insulator 30, and coils 31.

The stator core 28 is disposed more radially outward than the rotor 27; i.e. the stator core 28 radially surrounds the 35 rotor 27. The stator core 28 comprises a plurality of laminated steel sheets. Each of the steel sheets is a sheet made of a metal in which iron is the main component. The stator core 28 has a tube shape. The stator core 28 comprises a plurality of teeth that supports the coils 31.

The front insulator 29 is provided at a front portion of the stator core 28. The rear insulator 30 is provided at a rear portion of the stator core 28. The front insulator 29 and the rear insulator 30 are each an electrically insulating member that is made of a synthetic resin (polymer). The front 45 insulator 29 is disposed such that it covers a portion of the surface of each of the teeth. The rear insulator 30 is disposed such that it covers a portion of the surface of each of the teeth.

The coils 31 are mounted on the stator core 28 via the 50 front insulator 29 and the rear insulator 30. A plurality of the coils 31 is disposed. The coils 31 are respectively disposed around the teeth of the stator core 28 via (over) the front insulator 29 and the rear insulator 30. The coils 31 and the stator core 28 are electrically insulated from each other by 55 the front insulator 29 and the rear insulator 30. Pairs of the coils 31 are respectively electrically connected via bus bars.

The rotor 27 rotates about rotational axis AX. The rotor 27 comprises a rotor core 32, a rotor shaft 33, and rotor magnets 34

The rotor core 32 and the rotor shaft 33 are each made of steel. A front portion of the rotor shaft 33 protrudes forward from a front-end surface of the rotor core 32. A rear portion of the rotor shaft 33 protrudes rearward from a rear-end surface of the rotor core 32. The front portion and the rear portion of the rotor shaft 33 are each supported in a rotatable manner by rotor bearings 39. The rotor bearing 39 on the

6

front side is held by the bearing box 24. The rotor bearing 39 on the rear side is held by the rear-cover part 21B. A front-end portion of the rotor shaft 33 is disposed in the internal space of the hammer case 4 through an opening in the bearing box 24. The rotor magnets 34 are fixed to the rotor core 32.

A sensor board 37 is mounted on the rear insulator 30. The sensor board 37 comprises: a circuit board, which has a disk shape wherein a hole is provided at the center; and a rotation-detection device, which is supported by the circuit board. The rotation-detection device detects the position of the rotor 27 in the rotational direction by detecting the positions of the rotor magnets 34.

A pinion gear 41 is formed at a front-end portion of the rotor shaft 33. The pinion gear 41 is coupled to at least a portion of the speed-reducing mechanism 7. The rotor shaft 33 is coupled to the speed-reducing mechanism 7 via the pinion gear 41.

The speed-reducing mechanism 7 couples the rotor shaft 33 and the spindle 8 to each other. The speed-reducing mechanism 7 transmits the rotation of the rotor 27 to the spindle 8. The speed-reducing mechanism 7 causes the spindle 8 to rotate at a rotational speed that is lower than the rotational speed of the rotor 27, but at higher torque. The speed-reducing mechanism 7 comprises a planetary-gear mechanism. The speed-reducing mechanism 7 is disposed more forward than the motor 6.

The speed-reducing mechanism 7 comprises a plurality of planet gears 42 disposed around the pinion gear 41 and an internal gear 43 disposed around the plurality of planet gears 42. The pinion gear 41, the planet gears 42, and the internal gear 43 are each housed in the hammer case 4 and the bearing box 24. Each of the planet gears 42 meshes with the pinion gear 41. The planet gears 42 are supported in a rotatable manner by the spindle 8 via pins 42P. The spindle 8 is rotated by the planet gears 42. The internal gear 43 has inner teeth, which mesh with the planet gears 42. The internal gear 43 is fixed to the bearing box 24. The internal gear 43 is always non-rotatable relative to the bearing box 24.

When the rotor shaft 33 rotates in response to the operation (energization) of the motor 6, the pinion gear 41 rotates, and the planet gears 42 revolve around the pinion gear 41. The planet gears 42 revolve while meshing with the inner teeth of the internal gear 43. In response to the revolving of the planet gears 42, the spindle 8, which is connected to the planet gears 42 via the respective pins 42P, rotates at a rotational speed that is lower than the rotational speed of the rotor shaft 33.

The spindle 8 is rotated by the motor 6. The spindle 8 rotates in response to the rotational force of the rotor 27 transmitted by the speed-reducing mechanism 7. The spindle 8 is housed in the hammer case 4. The spindle 8 is disposed forward of the motor 6. At least a portion of the spindle 8 is disposed forward of the speed-reducing mechanism 7.

The spindle 8 comprises a spindle-shaft part 8A and a flange part 8B, which is disposed at a rear portion of the spindle-shaft part 8A. The spindle-shaft part 8A protrudes forward from the flange part 8B. The planet gears 42 are supported in a rotatable manner by the flange part 8B via the respective pins 42P. The rotational axis of the spindle 8 and rotational axis AX of the motor 6 coincide with each other. The spindle 8 rotates about rotational axis AX.

The spindle **8** is supported in a rotatable manner by a spindle bearing **44**. The spindle bearing **44** is held by the bearing box **24**.

The impact mechanism 9 impacts the anvil 10 in the rotational direction in response to transmission of the rotational force of the spindle 8. The rotational force of the motor 6 is transmitted to the impact mechanism 9 via the speed-reducing mechanism 7 and the spindle 8. The impact mechanism 9 comprises a hammer 47, balls 48, and a coil spring 49. The impact mechanism 9, which comprises the hammer 47, is housed in the hammer case 4.

The hammer **47** is configured to impact (strike) the anvil **10** in the rotational direction, as will be further described below. The hammer **47** is held on the spindle **8**. The hammer **47** is disposed around the spindle-shaft part **8A**. The balls **48** are disposed between the spindle **8** and the hammer **47**. The coil spring **49** is supported by both the spindle **8** and the hammer **47**.

The hammer 47 is rotatable, together with the spindle 8, in response to transmission of the rotational force of the spindle 8. The rotational axis of the hammer 47, the rotational axis of the spindle 8, and rotational axis AX of the 20 motor 6 coincide with each other. The hammer 47 rotates about rotational axis AX.

Each of the balls 48 is made of a metal such as steel. The balls 48 are disposed between the spindle-shaft part 8A and the hammer 47. The spindle 8 has spindle grooves, in which 25 at least portions of the balls 48 are respectively disposed. The spindle grooves are provided in portions of an outer surface of the spindle-shaft part 8A. The hammer 47 has hammer grooves, in which at least portions of the balls 48 are respectively disposed. The hammer grooves are provided 30 in portions of an inner surface of the hammer 47. The balls 48 are respectively disposed between the spindle grooves and the hammer grooves. The balls 48 can respectively roll in each of the interiors of the spindle grooves and the interiors of the hammer grooves. The hammer 47 is capable 35 of moving along with the balls 48. The spindle 8 and the hammer 47 are capable of relative movement in both the axial direction and the rotational direction within a movable range that is defined by the spindle grooves and the hammer

The coil spring 49 generates an elastic restoring force, which causes (biases) the hammer 47 to move forward. The coil spring 49 is disposed between the flange part 8B and the hammer 47. A recessed portion is provided on a rear surface of the hammer 47. The recessed portion recesses forward 45 from a rear surface of the hammer 47. A washer 45 is provided in the interior of the recessed portion. A rear-end portion of the coil spring 49 is supported by the flange part 8B. A front-end portion of the coil spring 49 is supported by the washer 45.

The anvil 10 is the output part of the impact tool 1, on which a tool accessory is mounted. The anvil 10 is disposed forward of the spindle 8. The anvil 10 is connected to a front-end portion of the spindle-shaft part 8A. At least a portion of the anvil 10 is disposed forward of the hammer 55 47. The anvil 10 comprises a quadrangular-column part 10C, on which a socket, which is a tool accessory, is mounted.

The anvil 10 comprises an anvil-shaft part 10A and anvil-projection parts 10B. The quadrangular-column part 10C is provided at a front-end portion of the anvil-shaft part 60 10A. The anvil-projection parts 10B protrude radially outward from a rear-end portion of the anvil-shaft part 10A.

The anvil 10 is supported in a rotatable manner by a bearing 46. The rotational axis of the anvil 10, the rotational axis of the hammer 47, the rotational axis of the spindle 8, 65 and rotational axis AX of the motor 6 coincide with each other. The anvil 10 rotates about rotational axis AX. The

8

bearing **46** is held by the hammer case **4**. An iron sleeve (e.g., made of an oil-impregnated metal) is an illustrative example of the bearing **46**.

At least portions of the hammer 47 are capable of making contact with the anvil-projection parts 10B. Hammer-projection parts, which protrude forward, are provided at a front portion of the hammer 47. The hammer-projection parts of the hammer 47 and the anvil-projection parts 10B are capable of making contact with each another. In the state in which the hammer 47 and the anvil-projection parts 10B are in contact with each another, the anvil 10 rotates together with the hammer 47 and the spindle 8 in response to the operation (energization) of the motor 6.

However, the anvil 10 is always configured to be impacted (struck) in the rotational direction by the hammer 47. For example, during screw-tightening work, there are situations in which, when the load that is acting on the anvil 10 becomes high (exceeds a predetermined threshold), the anvil 10 can no longer be caused to rotate merely by the power generated by the motor 6. Instead, when the anvil 10 can no longer be caused to rotate merely by the power generated by the motor 6, rotation of the anvil 10 and the hammer 47 will temporarily stop. The spindle 8 and the hammer 47 can move relative to each another in the axial direction and the circumferential direction via the balls 48. Even though rotation of the hammer 47 temporarily stops, rotation of the spindle 8 continues owing to the power generated by the motor 6. In the state in which rotation of the hammer 47 has temporarily stopped but the spindle 8 continues to rotate, the balls 48 move rearward while being guided by the respective spindle grooves and hammer grooves. The hammer 47 receives a force from the balls 48 and moves rearward along with the balls 48. That is, in the state in which the rotation of the anvil 10 is temporarily stopped, the hammer 47 moves rearward in response to the rotation of the spindle 8. The contact between the hammer 47 and the anvil-projection parts 10B is released by the movement of the hammer 47 rearward.

As was noted above, because the coil spring 49 generates an elastic restoring force, it causes (urges) the hammer 47 to move forward. Therefore, after the hammer 47 has been caused to move rearward, it will subsequently move forward owing to the elastic force of the coil spring 49. When the hammer 47 moves forward, it receives a force in the rotational direction from the balls 48. That is, the hammer 47 moves forward while rotating. When the hammer 47 moves forward while rotating, the hammer 47 makes contact with (impacts, strikes) the anvil-projection parts 10B while rotating. Thereby, the anvil-projection parts 10B are impacted in the rotational direction by the hammer 47. At this time, both the power of the motor 6 and the inertial force of the hammer 47 act on the anvil 10. As a result, the anvil 10 can be caused to rotate about rotational axis AX with a higher torque.

The fan 12 generates an airflow for cooling the motor 6. The fan 12 is fixed to a front portion of the rotor shaft 33. The fan 12 rotates in response to the rotation of the rotor 27. In other words, in response to the rotor shaft 33 rotating, the fan 12 rotates together with the rotor shaft 33. The motor-housing part 21 is provided with both air-intake openings 19 and air-exhaust openings 20. In response to the rotation of the fan 12, air in the external space of the housing 2 flows into the internal space of the housing 2 via the air-intake openings 19. The air that has flowed into the internal space of the housing 2, and thereby cools the motor 6. In response to the rotation of the fan 12, the air that has flowed through the

internal space of the housing 2 flows out to the external space of the housing 2 via the air-exhaust openings 20.

The battery-mounting part 13 is connected to the battery pack 25. The battery pack 25 is mounted on the battery-mounting part 13. The battery pack 25 is detachable from the battery-mounting part 13. The battery-mounting part 13 is disposed at a lower portion of the battery-holding part 23. The battery pack 25 is mounted on the battery-holding part 23, which is a portion of the housing 2, via the battery-mounting part 13.

The battery pack 25 comprises secondary batteries. In the embodiment, the battery pack 25 comprises rechargeable lithium-ion batteries. By being mounted on the battery-mounting part 13, the battery pack 25 can supply electric power (current) to the impact tool 1. The motor 6 operates (is energized) using electric power supplied from the battery pack 25. The controller 17 operates (is powered) using electric power supplied from the battery pack 25.

In the embodiment, the rated voltage of the battery pack $\ _{20}$ 25 is 18 V.

The trigger switch 14 is manipulated (e.g., pressed) by the user in order to start (the energization of) the motor 6. The trigger switch 14 is provided on the grip part 22. The trigger switch 14 comprises a trigger lever 14A and a switch main 25 body 14B. The switch main body 14B is housed in the grip part 22. The trigger lever 14A protrudes forward from an upper portion of a front portion of the grip part 22. In response to the trigger lever 14A being manipulated by the user, the motor 6 is switched between operation (energization) and stoppage.

The forward/reverse-change lever 15 is manipulated (pressed, slid) by the user in order to change the rotational direction of the motor 6 from one of the forward-rotational direction and the reverse-rotational direction to the other. 35 The forward/reverse-change lever 15 is provided at an upper portion of the grip part 22. In response to the forward/reverse-change lever 15 being manipulated, the rotational direction of the motor 6 is changed from one of the forward-rotational direction and the reverse-rotational direction to 40 the other. In response to the rotational direction of the motor 6 being changed, the rotational direction of the spindle 8 is changed.

The action-mode-change switch 16 is manipulated by the user in order to change the control mode of the motor 6, e.g., 45 a sequence of motor rotational speeds. The action-mode-change switch 16 is provided on the battery-holding part 23.

The controller 17 outputs control signals that control at least (the energization of) the motor 6. The controller 17 is housed in the battery-holding part 23. The controller 17 50 comprises a circuit board on which a plurality of electronic components is mounted. A processor, such as a CPU (central processing unit); nonvolatile memory, such as ROM (read-only memory) and storage; volatile memory, such as RAM (random-access memory); transistors; and resistors are illustrative examples of the electronic components mounted on the circuit board.

Light Unit

FIG. 4 is an exploded, oblique view, viewed from the front, that shows the light unit 18 according to the embodiment. The light unit 18 emits illumination light. The light unit 18 illuminates the anvil 10 and the periphery of the anvil 10 with illumination light. The light unit 18 illuminates forward of the anvil 10 with illumination light. In addition, the light unit 18 illuminates the tool accessory mounted on 65 the anvil 10 and the periphery of the tool accessory with illumination light.

10

The light unit 18 is held on the hammer case 4. The light unit 18 is disposed at a front portion of the hammer case 4. The light unit 18 is disposed at least partially around the hammer case 4.

The hammer case 4 comprises a hammer-housing part 4A and a bearing-retaining part 4B. The hammer-housing part 4A has a tube shape. The hammer-housing part 4A is disposed around the impact mechanism 9. The hammer-housing part 4A houses at least the hammer 47. The bearing-retaining part 4B has a tube shape. The bearing-retaining part 4B is disposed more forward than the hammer-housing part 4A. The outer diameter of the bearing-retaining part 4B is smaller than the outer diameter of the hammer-housing part 4A. The bearing-retaining part 4B is disposed around the bearing 46. The bearing-retaining part 4B holds the bearing 46.

The light unit 18 is disposed around the bearing-retaining part 4B. A rear portion of the hammer-housing part 4A is housed in the motor-housing part 21.

The light unit 18 comprises light-emitting devices 60, a light circuit board 61, optical members 62, and a light cover 63

Each of the light-emitting devices **60** is a light source that emits illumination light. Light-emitting diodes (LEDs: light-emitting diodes) are illustrative examples of the light-emitting devices **60**.

The plurality of the light-emitting devices 60 is provided such that the light-emitting devices 60 are spaced apart around the anvil 10. The number of light-emitting devices 60 is, for example, two or more and eight or less. At least three of the light-emitting devices 60 may be provided. The number of light-emitting devices 60 may be, for example, three or more and six or less. In the embodiment, four of the light-emitting devices 60 are provided around the anvil 10.

The light circuit board 61 supports the plurality of lightemitting devices 60. The light circuit board 61 is disposed at least partially around the hammer case 4. In the embodiment, the light circuit board 61 is disposed partially around the hammer case 4. The light circuit board 61 is disposed partially around the bearing-retaining part 4B.

The light circuit board 61 comprises a printed wiring board (PWB) or printed circuit board (PCB). The light circuit board 61 has wiring (traces, conductive paths) that is (are) connected to the light-emitting devices 60. Electric power (current) is supplied to the light-emitting devices 60 via the wiring of the light circuit board 61. The light-emitting devices 60 are mounted on a front surface of the light circuit board 61. In the embodiment, the light unit 18 comprises surface-mount-type (SMD: surface-mount device) light-emitting diodes. Each of the light-emitting devices 60 comprises a so-called chip LED.

The voltage input to each one of the light-emitting devices 60 is 1.0 volt (V) or more and 10.0 V or less. The voltage applied to each one of the light-emitting devices 60 may be, for example, 2.0 V or more and 8.0 V or less or may be 2.5 V or more and 5.0 V or less.

Electric current supplied to each one of the light-emitting devices 60 is 5 milliamps (mA) or more and 100 mA or less. Electric current supplied to each one of the light-emitting devices 60 may be 10 mA or more and 50 mA or less or may be 15 mA or more and 30 mA or less.

The light beam of the illumination light emitted from each one of the light-emitting devices 60 is 1 lumen (lm) or more and 20 lm or less. The light beam of the illumination light emitted from each one of the light-emitting devices 60 may be 3 lm or more and 15 lm or less or may be 5 lm or more and 10 lm or less.

The luminous intensity of the illumination light emitted from each one of the light-emitting devices 60 is 0.5 candela (cd) or more and 10 cd or less. The luminous intensity of the illumination light emitted from each one of the light-emitting devices 60 may be 1 cd or more and 7 cd or less or may 5 be 2 cd or more and 5 cd or less.

As shown in FIG. 4, the outer shape of each one of the light-emitting devices 60 is substantially rectangular-parallelepiped-shaped.

Width W of each one of the light-emitting devices **60** is 10 0.5 millimeters (mm) or more and 3 mm or less. Width W of each one of the light-emitting devices **60** may be 1 mm or more and 2 mm or less or may be 1.2 mm or more and 1.8 mm or less.

Length L of each one of the light-emitting devices **60** is 15 1.5 mm or more and 6 mm or less. Length L of each one of the light-emitting devices **60** may be 2.5 mm or more and 3.5 mm or less.

Thickness H of each one of the light-emitting devices **60** is 0.2 mm or more and 2 mm or less. Thickness H of each 20 one of the light-emitting devices **60** may be 0.3 mm or more and 1 mm or less or may be 0.4 mm or more and 0.8 mm or less.

The optical members 62 are disposed forward of the light-emitting devices 60 and the light circuit board 61. Each 25 of the optical members 62 comprises: light-transmitting parts 62A, which transmits the illumination light emitted from the corresponding light-emitting devices 60; and a coupling part 62B, which is connected to the light-transmitting parts 62A.

In the embodiment, the optical members 62 comprise an optical member 62L, which is disposed more leftward than rotational axis AX, and an optical member 62R, which is disposed more rightward than rotational axis AX. As can be seen, e.g., in FIG. 4, the optical members 62L, 62R do not 35 directly contact each other when integrated into the light cover 63, but are indirectly connected (linked) to each other via the material of the light cover 63 in the integrated state. The optical member 62L comprises two of the light-transmitting parts 62A. The optical member 62R comprises two 40 of the light-transmitting parts 62A. Of the four light-emitting devices 60, the two light-emitting devices 60 disposed more leftward than rotational axis AX respectively oppose the two light-transmitting parts 62A of the optical member 62L. Of the four light-emitting devices 60, the two light- 45 emitting devices 60 disposed more rightward than rotational axis AX respectively oppose the two light-transmitting parts 62A of the optical member 62R.

Each of the optical members **62** is formed of an optically transmissive synthetic resin (polymer). In the embodiment, 50 each of the optical members **62** is formed of a polycarbonate resin (polymer). It is noted that each of the optical members **62** may be formed of an acrylic resin (polymer).

Each of the light-transmitting parts **62A** has a lens function. Each of the light-transmitting parts **62A** refracts illumination light emitted from the corresponding light-emitting device **60**. It is noted that each of the light-transmitting parts **62A** does not have to have a lens function.

The light cover **63** is disposed forward of the lightemitting devices **60** and the light circuit board **61**. In the 60 embodiment, the light cover **63** is substantially ring-shaped.

The light cover 63 is formed of a synthetic resin (polymer). The light cover 63 may be formed of a material the same as that of the optical members 62. The light cover 63 may be formed of a material that differs from that of the 65 optical members 62. In the embodiment, the light cover 63 is formed of a polycarbonate resin (polymer). It is noted that

12

the light cover 63 may be formed of an acrylic resin (polymer). The optical members 62 and the light cover 63 are integrally molded. The optical members 62 and the light cover 63 are integrated by, for example, insert molding.

As can be seen in FIG. 4, openings 63A are provided spaced part in different portions of the light cover 63. The light-transmitting parts 62A of the optical members 62 are respectively disposed in the openings 63A of the light cover 63 such that the light-transmitting parts 62A are separated from each other on the outer surface of the light cover 63. The light-transmitting parts 62A are not covered by the light cover 63. That is, the light cover 63 is not disposed forward or rearward of the light-transmitting parts 62A. The coupling parts 62B of the optical members 62 are fixed to the light cover 63.

The optical members 62 and the light cover 63 are disposed around the bearing-retaining part 4B. The optical members 62 and the light cover 63 are supported on the hammer case 4 via the hammer-case cover 5.

The optical members 62 and the light cover 63 protect the light-emitting devices 60 and the light circuit board 61. The optical members 62 and the light cover 63 block contact between objects around the impact tool 1 on one side and the light-emitting devices 60 and the light circuit board 61 on the other side. The optical members 62 and the light cover 63 are integrally molded such that a gap is not formed between the optical members 62 and the light cover 63. The optical members 62 and the light cover 63 have a dustproofing function that inhibits the ingress of moisture to the light-emitting devices 60 and the light circuit board 61. The optical members 62 and the light cover 63 have a dustproofing function that inhibits the ingress of dust to the light-emitting devices 60 and the light circuit board 61.

Relationship Between Maximum Tightening Torque and Number of Light-Emitting Devices

FIG. **5** is a table that shows the specifications of a variety of known impact wrenches according. More particularly, FIG. **5** shows the specifications for: Product A, Product B, and Product C, which are impact wrenches manufactured and sold by Company α ; Product D, Product E, and Product F, which are impact wrenches manufactured and sold by Company β ; Product G, Product H, and Product I, which are impact wrenches manufactured and sold by Company γ ; and Product J, Product K, and Product L, which are impact wrenches manufactured and sold by Company δ . Each product from Product A to Product L has structural elements equivalent to the structural elements of the impact tool **1**, which was described above with reference to FIG. **1** to FIG. **4**. A battery pack is mounted on each product from Product A to Product L.

The number of light-emitting devices, the maximum tightening torque $[N \cdot m]$ of the anvil, the rated voltage [V] of the battery pack, the weight [kg] of the impact wrench in the state in which the battery pack is mounted, the overall length [mm] indicating the distance from the front-end portion of the anvil to the rear-end portion of the motor-housing part, the maximum rotational speed [rpm] of the anvil, and the wrench width [inches], which is defined as the dimension of one side of a cross section of the quadrangular-column part, are illustrative examples of the specifications of the impact wrench.

As shown in FIG. 5, the number of light-emitting devices is three for Product A, and similarly is three for Product B, one for Product C, three for Product D, one for Product E, three for Product F, one for Product G, three for Product H, one for Product I, one for Product J, one for Product K, and zero for Product L.

As shown in FIG. 5, the maximum tightening torque is 192 N m for Product A, and, expressed in a similar manner, is 339 N m for Product B, 949 N m for Product C, 2,576 N m for Product D, 2,034 N m for Product E, 339 N m for Product F, 678 N m for Product G, 115 N m for Product H, 181 N m for Product I, 100 N m for Product J, 260 N m for Product K, and 1.057 N m for Product L.

The values of the rated voltage (V) of the battery pack, the weight (kg) of the impact wrench in the state in which the battery pack is mounted, the overall length (mm), which is defined as the distance from the front-end portion of the anvil to the rear-end portion of the motor-housing part, the maximum rotational speed (rpm) of the anvil, and the wrench width (inches), which is defined as the dimension of one side of a cross section of the quadrangular-column part, are as shown in FIG. **5**.

In each product from Product A to Product L, the rated voltage of the battery pack is roughly 18 V.

FIG. **6** is a graph that shows the relationship between 20 maximum tightening torque of the anvil and number of light-emitting devices according to both the above-described known impact tools and the embodiment. In the graph shown in FIG. **6**, the abscissa is the number of light-emitting devices, and the ordinate is the maximum tightening torque 25 of the anvil. The points shown in FIG. **6** plot the relationship between maximum tightening torque of the anvil and number of the light-emitting devices for each product from Product A to Product L shown in FIG. **5**.

To provide an impact tool 1 having improved ergonomics 30 and/or work efficiency, it is effective to brightly illuminate the work environment with illumination light using the light unit 18. In addition, for the same purpose, it is effective to shorten the overall length. On the other hand, if the maximum tightening torque becomes large, there is a tendency 35 for the overall length of the impact tool 1 to become large. It is important to decide on a suitable tradeoff between overall length and maximum tightening torque of the impact tool 1

As described above, the impact tool 1 comprises a plurality of structural elements, such as the motor 6, the spindle 8, the impact mechanism 9, the anvil 10, and the light unit 18. By optimizing these structural elements, an improved impact tool 1 can be provided. In the present specification, the structural elements of the impact tool 1 are optimized, 45 and thereby the impact tool 1, in which ergonomics and/or work efficiency is (are) better than in the above-described known impact wrenches, is provided.

With regard to the impact tool 1 according to the embodiment, the battery pack 25 having a rated voltage of 18V is 50 mounted.

As shown by the hatched area in FIG. 6, the impact tool 1 according to the embodiment comprises the plurality of light-emitting devices 60, and the maximum tightening torque of the anvil 10 is 1,000 N·m or more and 2,500 N·m 55 or less. An impact wrench comprising a plurality of light-emitting devices and wherein the maximum tightening torque of the anvil 10 is 1,000 N·m or more and 2,500 N·m or less does not exist in the above-described known impact tools.

Relationship Between Weight and Number of Light-Emitting Devices

In addition, to provide an impact tool 1 having improved ergonomics and/or work efficiency, it is effective to optimize the weight of the impact tool 1. In the embodiment, the 65 weight of the impact tool is the weight of the impact tool in the state in which the battery pack is mounted.

14

FIG. 7 is a graph that shows the relationship between weight of the impact tool and number of the light-emitting devices according to both the above-described known impact tools and the embodiment. In the graph shown in FIG. 7, the abscissa is the number of light-emitting devices, and the ordinate is the weight of the impact tool. The points shown in FIG. 7 plot the relationship between the weight of the impact tool and the number of the light-emitting devices for each product from Product A to Product L shown in FIG. 5

As shown by the hatched area in FIG. 7, the impact tool 1 according to the embodiment comprises at least three of the light-emitting devices 60 and has a weight of 2 kg or more and 9 kg or less. An impact wrench comprising at least three of the light-emitting devices and having a weight of 2 kg or more and 9 kg or less does not exist in the above-described known impact tools.

It is noted that, in the impact tool 1 comprising the plurality of light-emitting devices 60 and wherein the maximum tightening torque of the anvil 10 is 1,000 N·m or more and 2,500 N·m or less, the weight of the impact tool 1 may be 2 kg or less.

Relationship Between Overall Length and Number of Light-Emitting Devices

In addition, to provide an impact tool 1 having improved ergonomics and/or work efficiency, it is effective to optimize the overall length of the impact tool 1. As shown in FIG. 1, overall length La of the impact tool 1 is the distance from a front-end portion of the anvil 10 to a rear-end portion of the motor-housing part 21.

FIG. 8 is a graph that shows the relationship between overall length of the impact tool and number of the light-emitting devices according to both the above-described known impact tools and the embodiment. In the graph shown in FIG. 8, the abscissa is the number of light-emitting devices, and the ordinate is the overall length of the impact tool. The points shown in FIG. 8 plot the relationship between overall length of the impact tool and number of the light-emitting devices for each product from Product A to Product L shown in FIG. 5.

As shown by the hatched area in FIG. 8, the impact tool 1 according to the embodiment comprises at least three of the light-emitting devices 60, and has overall length La of 200 mm or more and 400 mm or less. An impact wrench comprising at least three of the light-emitting devices and having an overall length of 200 mm or more and 400 mm or less does not exist in the above-described known impact tools.

It is noted that, in the impact tool 1 comprising the plurality of light-emitting devices 60 and wherein the maximum tightening torque of the anvil 10 is 1,000 N·m or more and 2,500 N·m or less, overall length La of the impact tool 1 may be 155 mm or less.

Relationship Between Maximum Rotational Speed of Anvil and Number of Light-Emitting Devices

In addition, to provide an impact tool 1 having improved ergonomics and/or work efficiency, it is effective to optimize the maximum rotational speed of the anvil 10.

FIG. 9 is a graph that shows the relationship between maximum rotational speed of the anvil and number of the light-emitting devices according to both the above-described known impact tools and the embodiment. In the graph shown in FIG. 9, the abscissa is the number of light-emitting devices, and the ordinate is the maximum rotational speed of the anvil. The points shown in FIG. 9 plot the relationship between maximum rotational speed of the anvil and number

of the light-emitting devices for each product from Product A to Product L shown in FIG. 5.

As shown by the hatched area in FIG. 9, the impact tool 1 according to the embodiment comprises at least three of the light-emitting devices 60 and wherein the maximum 5 rotational speed of the anvil 10 is 1,300 rpm or more and 2,300 rpm or less. An impact wrench comprising at least three of the light-emitting devices and wherein the maximum rotational speed is 1,300 rpm or more and 2,300 rpm or less does not exist in the above-described known impact 10 tools.

It is noted that, in the impact tool 1 comprising the plurality of light-emitting devices 60 and wherein the maximum tightening torque of the anvil 10 is 1,000 N·m or more and 2,500 N·m or less, the maximum rotational speed of the 15 anvil 10 may be 3,000 rpm or less.

Relationship Between Wrench Width of Anvil and Number of Light-Emitting Devices In addition, to provide an impact tool 1 having improved ergonomics and/or work efficiency, it is effective to optimize the wrench width of the 20 anvil 10. As shown in FIG. 2, wrench width W1 of the anvil 10 is the dimension of one side of a cross section of the quadrangular-column part 10C.

FIG. 10 is a graph that shows the relationship between wrench width of the anvil and number of the light-emitting devices according to both the above-described known impact tools and the embodiment. In the graph shown in FIG. 10, the abscissa is the number of light-emitting devices, and the ordinate is the wrench width of the anvil. The points shown in FIG. 10 plot the relationship between wrench width of the anvil and number of the light-emitting devices for each product from Product A to Product L shown in FIG. 5.

As shown by the hatched area in FIG. 10, the impact tool 1 according to the embodiment comprises at least three of 35 the light-emitting devices 60 and wherein the wrench width W1 of the anvil 10 is 0.6 inches or more and 0.9 inches or less. An impact wrench comprising at least three of the light-emitting devices and wherein the wrench width is 0.6 inches or more and 0.9 inches or less does not exist in the 40 above-described known impact tools.

It is noted that, in the impact tool 1 comprising the plurality of light-emitting devices 60 and wherein the maximum tightening torque of the anvil 10 is $1,000 \text{ N} \cdot \text{m}$ or more and $2,500 \text{ N} \cdot \text{m}$ or less, wrench width W1 may be 0.375 45 inches or less.

Effects

As explained above, an impact tool 1 comprises: a motor 50 6; a spindle 8, which is rotated by the motor 6; a hammer 47, which is held on the spindle 8; an anvil 10, which is impacted in a rotational direction by the hammer 47; a housing 2, which houses the motor 6; and a hammer case 4, which is connected to the housing 2 and houses the hammer 55 47 and the spindle 8. The impact tool 1 comprises a light unit 18, which is held on the hammer case 4 and comprises a plurality of light-emitting devices 60. The maximum tightening torque of the anvil 10 is 1,000 N·m or more and 2,500 N·m or less.

According to the above-mentioned configuration, because the light unit 18 comprises the plurality of light-emitting devices $60,\,$ the work environment is brightly illuminated with illumination light. In addition, the maximum tightening torque of the anvil 10 is $1,\!000~\rm N\cdot m$ or more and $2,\!500~\rm N\cdot m$ $_{65}$ or less. Consequently, an impact tool 1 having improved ergonomics and/or work efficiency is provided.

16

In the embodiment, at least three of the light-emitting devices 60 are provided.

According to the above-mentioned configuration, because the light unit 18 comprises at least three of the light-emitting devices 60, the work environment is brightly illuminated with illumination light. Consequently, an impact tool 1 having improved ergonomics and/or work efficiency is provided.

In the embodiment, a battery pack 25 is mounted on a battery-holding part 23, which is a portion of the housing 2. The weight of the impact tool 1 is 2 kg or less.

According to the above-mentioned configuration, because the impact tool 1, with the battery pack 25 mounted thereon, has a weight of 2 kg or less, an impact tool 1 having improved ergonomics and/or work efficiency is provided.

In the embodiment, a battery pack 25 is mounted on a battery-holding part 23, which is a portion of the housing 2. At least three of the light-emitting devices 60 are provided. The weight of the impact tool 1 is 2 kg or more and 9 kg or less.

According to the above-mentioned configuration, because the light unit 18 comprises at least three of the light-emitting devices 60 and the impact tool 1, with the battery pack 25 mounted thereon, has a weight of 2 kg or more and 9 kg or less, an impact tool 1 having improved ergonomics and/or work efficiency is provided.

In the embodiment, the spindle 8 is disposed forward of the motor 6; the anvil 10 is disposed forward of the spindle 8; and the motor 6 is housed in a motor-housing part 21, which is a portion of the housing 2. The overall distance La, which is defined as the distance from a front-end portion of the anvil 10 to a rear-end portion of the motor-housing part 21, is 155 mm or less.

According to the above-mentioned configuration, because the overall length La of the impact tool 1, which is defined as the distance between the front-end portion of the anvil 10 and the rear-end portion of the motor-housing part 21, is 155 mm or less, an impact tool 1 having improved ergonomics and/or work efficiency is provided.

In the embodiment, the spindle 8 is disposed forward of the motor 6; the anvil 10 is disposed forward of the spindle 8; and the motor 6 is housed in a motor-housing part 21, which is a portion of the housing 2. At least three of the light-emitting devices 60 are provided. The overall distance La, which is defined as the distance from a front-end portion of the anvil 10 to a rear-end portion of the motor-housing part 21, is 200 mm or more and 400 mm or less.

According to the above-mentioned configuration, because the light unit 18 comprises at least three of the light-emitting devices 60 and the overall length La of the impact tool 1, which is defined as the distance from the front-end portion of the anvil 10 to the rear-end portion of the motor-housing part 21, is 200 mm or more and 400 mm or less, an impact tool 1 having improved ergonomics and/or work efficiency is provided.

In the embodiment, the maximum rotational speed of the anvil 10 is 3,000 rpm or less.

According to the above-mentioned configuration, because the maximum rotational speed of the anvil 10 is 3,000 rpm or less, an impact tool 1 having improved ergonomics and/or work efficiency is provided.

In the embodiment, at least three of the light-emitting devices **60** are provided. The maximum rotational speed of the anvil **10** is 1,300 rpm or more and 2,300 rpm or less.

According to the above-mentioned configuration, because the light unit 18 comprises at least three of the light-emitting devices 60 and the maximum rotational speed of the anvil 10

is 1,300 rpm or more and 2,300 rpm or less, an impact tool 1 having improved ergonomics and/or work efficiency is provided.

In the embodiment, the anvil **10** has a quadrangular-column part **10**C. The wrench width **W1**, which is defined as the dimension of one side of a cross section of the quadrangular-column part **10**C, is 0.375 inches or less.

According to the above-mentioned configuration, because the wrench width W1, which is defined as the dimension of one side of a cross section of the quadrangular-column part 10C, is 0.375 inches or less, an impact tool 1 having improved ergonomics and/or work efficiency is provided.

In one or more embodiments, the anvil 10 has a quadrangular-column part 10C. At least three of the light-emitting devices 60 are provided. The wrench width W1, which is defined as the dimension of one side of a cross section of the quadrangular-column part 10C, is 0.6 inches or more and 0.9 inches or less.

According to the above-mentioned configuration, because the light unit **18** comprises at least three of the light-emitting devices **60** and the wrench width W**1**, which is defined as the dimension of one side of a cross section of the quadrangular-column part **10**C, is 0.6 inches or more and 0.9 inches or less, an impact tool **1** having improved ergonomics and/or work efficiency is provided.

In the embodiment, the light unit 18 comprises: a light circuit board 61, which is disposed at least partially around the hammer case 4 and holds a plurality of the light-emitting devices 60; and optical members 62, which are disposed forward of the light-emitting devices 60 and the light circuit 30 board 61.

According to the above-mentioned configuration, the light-emitting devices 60 and the light circuit board 61 are protected by the optical members 62.

Modified Examples

FIGS. 11-13 schematically show modified examples of the light circuit board 61 according to the above-described embodiment. As shown in FIG. 11, a light circuit board 61A 40 may have a ring shape. As shown in FIG. 12, a light circuit board 61B may have an arc shape. A gap 61G is provided between one-end portion and the other-end portion of the light circuit board 61B. As shown in FIG. 13, the gap 61G between one-end portion and the other-end portion of a light 45 circuit board 61C may be large.

In the embodiments described above, it is assumed that the impact tool 1 is an impact wrench. The impact tool 1 may be an impact driver.

In the embodiments described above, the power supply of 50 the impact tool 1 does not have to be the battery pack 25 and may be commercial power supply (AC power supply).

Representative, non-limiting examples of the present invention were described above in detail with reference to the attached drawings. This detailed description is merely 55 intended to teach a person of skill in the art further details for practicing preferred aspects of the present teachings and is not intended to limit the scope of the invention. Furthermore, each of the additional features and teachings disclosed above may be utilized separately or in conjunction with 60 other features and teachings to provide improved impact tools, such as impact wrenches and impact drivers.

Moreover, combinations of features and steps disclosed in the above detailed description may not be necessary to practice the invention in the broadest sense, and are instead 65 taught merely to particularly describe representative examples of the invention. Furthermore, various features of 18

the above-described representative examples, as well as the various independent and dependent claims below, may be combined in ways that are not specifically and explicitly enumerated in order to provide additional useful embodiments of the present teachings.

All features disclosed in the description and/or the claims are intended to be disclosed separately and independently from each other for the purpose of original written disclosure, as well as for the purpose of restricting the claimed subject matter, independent of the compositions of the features in the embodiments and/or the claims. In addition, all value ranges or indications of groups of entities are intended to disclose every possible intermediate value or intermediate entity for the purpose of original written disclosure, as well as for the purpose of restricting the claimed subject matter.

EXPLANATION OF THE REFERENCE NUMBERS

- 1 Impact tool
- **2** Housing
- 2L Left housing
- 2R Right housing
- 2S Screw
- 4 Hammer case
- 4A Hammer-housing part
- 4B Bearing-retaining part
- 5 Hammer-case cover
- 6 Motor
- 7 Speed-reducing mechanism
- 8 Spindle
- 8A Spindle-shaft part
- 8B Flange part
- 9 Impact mechanism
- 10 Anvil
- 10A Anvil-shaft part
- 10B Anvil-projection part
- 10C Quadrangular-column part
- **12** Fan
- 13 Battery-mounting part
- 14 Trigger switch
- 14A Trigger lever
- 14B Switch main body
- 15 Forward/reverse-change lever
- 16 Mode-change switch
- 17 Controller
- 18 Light unit
- 19 Air-intake opening
- 20 Air-exhaust opening
- 21 Motor-housing part
- 21A Tubular part
- 21B Rear-cover part
- 22 Grip part
- 23 Battery-holding part
- 24 Bearing box
- 25 Battery pack
- 26 Stator
- 27 Rotor
- 28 Stator core
- 29 Front insulator
- 30 Rear insulator
- 31 Coil
- 32 Rotor core
- 33 Rotor shaft
- 34 Rotor magnet
- 37 Sensor board

39 Rotor bearing

19

- 41 Pinion gear
- 42 Planet gear
- **42**P Pin
- 43 Internal gear
- 44 Spindle bearing
- 45 Washer
- **46** Bearing
- 47 Hammer
- 48 Ball
- 49 Coil spring
- 60 Light-emitting device
- 61 Light circuit board
- 61A Light circuit board
- 61B Light circuit board
- 610 Light circuit board
- 61G Gap
- 62 Optical member
- 62A Light-transmitting part
- 62B Coupling part
- 62L Optical member
- 62R Optical member
- 63 Light cover
- 63A Opening
- AX Rotational axis
- H Thickness
- L Length
- La Overall length
- W Width
- W1 Wrench width

We claim:

- 1. An impact tool comprising:
- a brushless motor, which includes a stator and a rotor configured to rotate relative to the stator;
- a spindle, which is rotated by the brushless motor;
- a hammer, which is held on the spindle,
- an anvil, which is impacted in a rotational direction by the hammer, the anvil having a quadrangular-column part and having a maximum tightening torque of 1,000 N·m 40 or more and 2,500 N·m or less;
- a polymer housing, which houses the brushless motor;
- a hammer case, which is connected to the polymer housing and houses the hammer and the spindle, the hammer case including a hammer-housing part that houses 45 the hammer and a bearing-retaining part that holds a bearing;
- a cover, which covers at least a portion of the hammer case:
- three or more optical members, which are held on the 50 cover:
- three or more light-emitting devices, which are respectively disposed rearward of the three or more optical members; and
- a light circuit board on which the three or more light- 55 emitting devises are fixed;

wherein:

- the three or more optical members respectively face the light-emitting devices and are configured to transmit illumination light emitted from the three or more light- 60 emitting devices; and
- the cover, the three or more optical members, the three or more light-emitting devices, and the light circuit board are all disposed around the bearing-retaining part and forward of a rear end of the anvil.
- 2. The impact tool according to claim 1, wherein the impact tool has a weight of 2 kg or more and 9 kg or less.

20

- 3. The impact tool according to claim 1, wherein a distance from a front end of the anvil to a rear end of the polymer housing is 200 mm or more and 400 mm or less.
 - 4. The impact tool according to claim 3, wherein:
- the three or more optical members are composed of a first material, and
 - the cover is composed of a second material that differs from the first material.
- 5. The impact tool according to claim 4, wherein the cover has three or more openings that are spaced apart in different portions of the cover such that portions of the three or more optical members are respectively disposed in the three or more openings.
- 6. The impact tool according to claim 1, wherein the anvil 15 has a maximum rotational speed of 1,300 rpm or more and 2,300 rpm or less.
 - 7. The impact tool according to claim 1, wherein:
 - the three or more optical members are composed of a first material, and
- the cover is composed of a second material that differs from the first material.
 - 8. The impact tool according to claim 7, wherein:
 - the three or more optical members do not directly contact each other; and
- the three or more optical members and the cover are integrated such that the three or more optical members are indirectly connected to each other via the second material.
- 9. The impact tool according to claim 1, wherein the cover 30 has three or more openings that are spaced apart in different portions of the cover such that portions of the three or more optical members are respectively disposed in the three or more openings.
- 10. The impact tool according to claim 1, wherein the bearing rotatably supports the anvil.
 - 11. The impact tool according to claim 10, wherein the cover is ring-shaped and radially surrounds the anvil.
 - 12. The impact tool according to claim 1, wherein the three or more optical members are optically independent from each other.
 - 13. An impact tool, comprising:
 - a brushless motor, which includes a stator and a rotor configured to rotate relative to the stator;
 - a spindle, which is rotated by the brushless motor;
 - a hammer, which is held on the spindle,
 - an anvil, which is impacted in a rotational direction by the hammer:
 - a polymer housing, which houses the brushless motor;
 - a hammer case, which is connected to the polymer housing and houses the hammer and the spindle;
 - a cover, which covers at least a portion of the hammer case;
 - three or more optical members, which are held on the cover; and
 - three or more light-emitting devices, which are respectively disposed rearward of the three or more optical member;
 - wherein: the three or more optical members respectively face the light-emitting devices and are configured to transmit illumination light emitted from the three or more light-emitting devices;
 - the three or more optical members are composed of a first material that comprises a polycarbonate or an acrylic,
 - the cover is composed of a second material that differs from the first material, and
 - the second material comprises a polycarbonate or an acrylic.

- 14. The impact tool according to claim 13, wherein the anvil has a maximum tightening torque of $1,000~\text{N}\cdot\text{m}$ or more and $2,500~\text{N}\cdot\text{m}$ or less.
- **15**. The impact tool according to claim **14**, wherein the cover has three or more openings that are spaced apart in different portions of the cover such that portions of the three or more optical members are respectively disposed in the three or more openings.
- 16. The impact tool according to claim 15, wherein the impact tool has a weight of 2 kg or more and 9 kg or less.
- 17. The impact tool according to claim 16, wherein a distance from a front end of the anvil to a rear end of the polymer housing is 200 mm or more and 400 mm or less.
 - **18**. The impact tool according to claim **17**, wherein: the three or more optical members do not directly contact each other; and
 - the three or more optical members and the cover are integrated such that the three or more optical members are indirectly connected to each other via the second material.
 - 19. An impact tool comprising:
 - a brushless motor, which includes a stator and a rotor configured to rotate relative to the stator;
 - a spindle, which is rotated by the brushless motor;
 - a hammer, which is held on the spindle,
 - an anvil, which is impacted in a rotational direction by the hammer, the anvil having a quadrangular-column part and having a maximum tightening torque of 1,000 N·m or more and 2,500 N·m or less;
 - a polymer housing, which houses the brushless motor;
 - a hammer case, which is connected to the polymer housing;

22

- a cover, which covers at least a portion of the hammer case:
- three or more optical members, which are held on the cover:
- three or more light-emitting devices, which are respectively disposed rearward of the three or more optical members; and
- a light circuit board on which the three or more lightemitting devises are fixed;

wherein:

- the three or more optical members respectively face the light-emitting devices and are configured to transmit illumination light emitted from the three or more lightemitting devices, and
- the cover and the three or more optical members are integrally molded such that a gap is not formed therebetween, so as to inhibit ingress of moisture or dust.
- 20. The impact tool according to claim 19, wherein the cover is ring-shaped and radially surrounds the anvil.
 - 21. The impact tool according to claim 20, wherein: the three or more optical members are composed of a first material, and
 - the cover is composed of a second material that differs from the first material.
- 22. The impact tool according to claim 21, wherein: the three or more optical members do not directly contact each other; and
- the three or more optical members and the cover are integrated such that the three or more optical members are indirectly connected to each other via the second material.

* * * * *