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TURNING CONTROL DEVICE AND TURNING DEVICE

Abstract

A turning control device includes: a terminal position learning unit configured to learn a terminal position of the turning mechanism, based on a steered position of a turning mechanism detected by the position detection unit; and a relearning determination unit configured to determine necessity of relearning of the terminal position. The relearning determination unit includes: an end-abutting detection unit configured to detect occurrence of end-abutting, and acquire an end-abutting steering angle that is a steering angle when end-abutting is occurred; an end-abutting steering angle range determination unit configured to determine whether or not variation in the end-abutting steering angles acquired multiple times is less than or equal to a predetermined threshold value; and a relearning unit configured to, when the variation is less than or equal to the predetermined threshold value, reset the learned terminal position to an initial value.

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Background/Summary

CROSS REFERENCE TO RELATED APPLICATIONS [0001] This application is a National Stage of International Application No. PCT/JP2024/021687 filed Jun. 14, 2024, claiming priority based on Japanese Patent Application No. 2023-121872 filed Jul. 26, 2023.

TECHNICAL FIELD

[0002] The present invention relates to a turning control device and a turning device.

BACKGROUND ART

[0003] In a turning mechanism of a vehicle, when a turning angle increases and reaches a maximum mechanical turning angle, a rack shaft of the turning mechanism reaches a stroke end and it becomes impossible to increase the turning angle any further. The rack shaft being brought to a state of having reached a stroke end as described above is referred to as "end-abutting". In addition, a stroke end of the rack shaft is sometimes referred to as "rack end".

[0004] When end-abutting occurs at a high turning velocity, there is a risk that large impact and hit sound (abnormal noise) are generated and the driver feels uncomfortable. In PTLs 1 and 2, technologies for mitigating impact at the time of end-abutting by learning a maximum value of an absolute value of a steering angle as rack end positions and preventing increase in a steering angle when the steering angle detected by a sensor is in a vicinity of one of the learned rack end positions are described.

[0005] Hereinafter, a virtual rack end position learned based on a steering angle detected by the sensor is sometimes referred to as "virtual rack end position", and an actual physical rack end position is sometimes referred to as "actual rack end position".

CITATION LIST

Patent Literature

[0006] PTL 1: JP 7131737 B [0007] PTL 2: JP 7136398 B

SUMMARY OF INVENTION

Technical Problem

[0008] There are some cases where after virtual rack end positions are learned, a rack shaft is replaced. On this occasion, it is conceivable that a rack shaft having a length different from length of a rack shaft that should be originally mounted is mistakenly mounted or a mounting position of a rack shaft changes between before and after replacement of the rack shaft. In such a case, it is necessary to relearn the virtual rack end positions. There is a risk that when the virtual rack end positions are relearned, an inappropriate virtual rack end position is learned due to a reason such as rapid turning-back steering and collision of a tire with a curb.

[0009] The present invention has been made in consideration of the above-described problem, and an object of the present invention is to prevent incorrect learning at the time of relearning of a rack end position.

Solution to Problem

[0010] In order to achieve the above-described object, according to an aspect of the present invention, there is provided a turning control device including: a position detection unit configured

to detect a steered position of a turning mechanism of a vehicle; a terminal position learning unit configured to learn a terminal position of the turning mechanism, based on the steered position detected by the position detection unit; and a relearning determination unit configured to determine necessity of relearning of the terminal position, wherein the relearning determination unit includes: an end-abutting detection unit configured to detect occurrence of end-abutting, the end-abutting being a state in which the turning mechanism is turned to the terminal position, and acquire an end-abutting steering angle, the end-abutting steering angle being a steering angle when occurrence of end-abutting is detected; an end-abutting steering angle range determination unit configured to determine whether or not variation in the end-abutting steering angles acquired multiple times is less than or equal to a predetermined threshold value; and a relearning unit configured to, when the variation is less than or equal to the predetermined threshold value, reset the learned terminal position to an initial value.

[0011] According to another aspect of the present invention, there is provided a turning device including: the turning control device described above; and an actuator configured to, drive-controlled by the turning control device, turn a steered wheel of the vehicle.

Advantageous Effects of Invention

[0012] According to the present invention, it is possible to prevent incorrect learning at the time of relearning of a rack end position.

Description

BRIEF DESCRIPTION OF DRAWINGS

[0013] FIG. **1** is a configuration diagram illustrative of an outline of an example of an electric power steering device of an embodiment;

[0014] FIG. **2** is a block diagram illustrative of an example of a functional configuration of a controller illustrated in FIG. **1**;

[0015] FIG. **3** is an explanatory diagram of an example of a steering angle range where impact mitigation control is performed;

[0016] FIG. **4** is a block diagram illustrative of an example of a functional configuration of an impact mitigation control unit;

[0017] FIGS. 5A and 5B are a characteristic diagram illustrative of a characteristic example of a spring constant table and a characteristic diagram illustrative of a characteristic example of a viscosity constant table, respectively;

[0018] FIG. **6** is a block diagram illustrative of an example of a functional configuration of a terminal position learning unit;

[0019] FIGS. 7A and 7B are an explanatory diagram of an example of change in column output shaft torque associated with change in a steering angle and an explanatory diagram of an example of a learned value of a terminal position when the column output shaft torque in FIG. 7A is generated, respectively;

[0020] FIGS. **8**A, **8**B, **8**C, **8**D, **8**E, and **8**F are a conceptual diagram of actual rack end positions, a conceptual diagram of a state before start of learning of virtual rack end positions, a conceptual diagram of a state in which a right virtual rack end position has been learned, a conceptual diagram of a state in which a left virtual rack end position has been learned, a conceptual diagram of a state in which learning of the virtual rack ends is considered to have been completed, and a conceptual diagram of a state in which the learning of the virtual rack ends has been continued to the vicinities of the actual rack ends, respectively;

[0021] FIGS. **9**A, **9**B, **9**C, **9**D, **9**E, **9**F, **9**G, and **9**H are a conceptual diagram of the actual rack end positions, a conceptual diagram illustrative of the same state as the state in FIG. **8**F, a conceptual diagram of a state immediately after offset error has occurred, a conceptual diagram of a state in

which a learned position of the left virtual rack end position is updated, a conceptual diagram of correction of a steering angle detected by a steering angle sensor and resetting of a learned value of the right virtual rack end position, a conceptual diagram of a state in which a learned position of the left virtual rack end position is further updated, a conceptual diagram of a state in which learning of the virtual rack ends is considered to have been completed, and a conceptual diagram of a state in which a learned position of the left virtual rack end position is further updated, respectively; [0022] FIG. **10** is a table illustrative of an example of limiting values in the states in FIGS. **8**B to **8**F and FIGS. **9**B to **9**H;

- [0023] FIG. **11** is a block diagram illustrative of an example of a functional configuration of a relearning determination unit;
- [0024] FIGS. **12**A and **12**B are an explanatory diagram of an example of end-abutting steering angle necessity determination and an explanatory diagram of an example of end-abutting steering angle acquisition possibility determination, respectively;
- [0025] FIGS. **13**A and **13**B are explanatory diagrams of examples of rack stroke determination; [0026] FIGS. **14**A to **14**C are explanatory diagrams of examples of end-abutting steering angle range determination;
- [0027] FIG. **15** is an example of a state transition diagram of a vehicle state;
- [0028] FIG. **16** is an explanatory diagram of an example of a method for calculating a rack stroke minimum value Stmin, a rack stroke maximum value Stmax, a learning threshold value θ lth, a rack end maximum value θ evmax, and an initial value θ int;
- [0029] FIG. **17** is a flowchart of an example of a turning control method of the embodiment; [0030] FIG. **18** is a flowchart of an example of one-side end-abutting processing in FIG. **17**; and [0031] FIG. **19** is a flowchart of an example of both-side end-abutting processing in FIG. **17**. DESCRIPTION OF EMBODIMENTS

[0032] Embodiments of the present invention will be described in detail with reference to the drawings. Note that the embodiments of the present invention to be described below indicate devices and methods to embody the technical idea of the present invention by way of example, and the technical idea of the present invention does not limit the constitution, arrangements, and the like of the constituent components to those described below. The technical idea of the present invention can be subjected to a variety of alterations within the technical scope prescribed by the claims described in CLAIMS.

(Configuration)

- [0033] FIG. **1** is a configuration diagram illustrative of an outline of an example of an electric power steering device of an embodiment. Column shafts (steering shafts) **2***i* and **20** of a steering wheel **1** is connected to steered wheels **8**L and **8**R by way of a reduction gear (worm gear) **3**, which constitutes a speed reduction mechanism, an intermediate shaft **4**, a pinion rack mechanism **5**, and tie rods **6***a* and **6***b* and further via hub units **7***a* and **7***b*.
- [0034] The column input shaft **21** and the column output shaft **20** are connected by a torsion bar (not illustrated) that is twisted due to a difference in rotation angles between the column input shaft **2***i* and the column output shaft **20**.
- [0035] The intermediate shaft **4** includes a shaft member **4***c* and universal joints **4***a* and **4***b* that are attached to both ends of the shaft member. The universal joint **4***a* is coupled to the column output shaft **20**, and the universal joint **4***b* is coupled to the pinion rack mechanism **5**.
- [0036] The pinion rack mechanism $\mathbf{5}$ includes a pinion $\mathbf{5}a$ that is coupled to a pinion shaft to which steering force is transmitted from the universal joint $\mathbf{4}b$ and a rack $\mathbf{5}b$ that meshes with the pinion $\mathbf{5}a$, and converts rotational motion transmitted to the pinion $\mathbf{5}a$ to linear motion in the vehicle width direction by the rack $\mathbf{5}b$.
- [0037] To the steering shaft **2** (column shafts **2**i and **2**o), a torque sensor **10** configured to detect steering torque Th is disposed. To the steering shaft **2** (column shafts **2**i and **2**o), a steering angle sensor **14** configured to detect a steering angle θ h of the steering wheel **1** is also disposed.

[0038] A motor **20** configured to assist steering force of the steering wheel **1** is also connected to the column output shaft **2***o* via the reduction gear **3**. To a controller **30** configured to control the electric power steering (EPS) device, power is supplied from a battery **13** and an ignition key signal is also input via an ignition (IGN) key **11**.

[0039] Note that a means for providing steering assist force is not limited to a motor and a variety of types of actuators can be used.

[0040] The controller **30** performs calculation of a current command value of an assist control command, based on steering torque Th detected by the torque sensor **10**, vehicle speed Vh detected by a vehicle speed sensor **12**, and a steering angle θ h detected by the steering angle sensor **14** and controls current to be supplied to the motor **20** by a voltage control command value Vref obtained by performing compensation and the like on the calculated current command value.

[0041] Note that the steering angle sensor **14** is not an essential component and the steering angle θ h may be calculated by adding a torsion angle of a torsion bar in the torque sensor **10** to a rotation angle obtained from a rotation angle sensor configured to detect a rotation angle of the rotation shaft of the motor **20**.

[0042] The controller **30** may include, for example, a computer including a processor and peripheral components, such as a storage device. The processor may be, for example, a central processing unit (CPU) or a micro-processing unit (MPU).

[0043] The storage device may include any one of a semiconductor storage device, a magnetic storage device, and an optical storage device. The storage device may include registers, a cache memory, or a memory, such as a read only memory (ROM) and a random access memory (RAM), that is used as a main storage device.

[0044] Functions of the controller **30**, which will be described below, are achieved by, for example, the processor of the controller **30** executing computer programs stored in the storage device. [0045] Note that the controller **30** may be formed by use of dedicated hardware for performing respective units of information processing, which will be described below. For example, the controller **30** may include functional logic circuits that are set in a general-purpose semiconductor integrated circuit. For example, the controller **30** may have a programmable logic device (PLD), such as a field-programmable gate array (FPGA), or the like.

[0046] FIG. **2** is a block diagram illustrative of an example of a functional configuration of the controller **30** of the embodiment. The controller **30** includes a basic command value calculation unit **40**, an adder **41**, a subtracter **42**, a current control unit **43**, a pulse width modulation (PWM) control unit **44**, an inverter (INV) **45**, a terminal position learning unit **46**, a control rotational displacement setting unit **47**, a differentiating unit **48**, an impact mitigation control unit **49**, a current detector **50**, a learning state determination unit **51**, an impact mitigation control output limiting unit **52**, and a relearning determination unit **53**.

[0047] The basic command value calculation unit **40** calculates a basic current command value Iref**1** that is a control target value of driving current of the motor **20**, based on the steering torque Th from the torque sensor **10** and the vehicle speed Vh from the vehicle speed sensor **12**. [0048] In the present embodiment, a value of the basic current command value Iref**1** that causes a steering assist force of the motor **20** to be generated in the rightward steering direction is defined as a positive value, and a value of a basic current command value Iref**1** that causes a steering assist force to be generated in the leftward steering direction is defined as a negative value.

[0049] The adder **41** corrects the basic current command value Iref**1** by adding impact mitigation control output Iref**2**′ output from the impact mitigation control output limiting unit **52** to the basic current command value Iref**1** and outputs the basic current command value Iref**1** after correction as a current command value Iref**3**.

[0050] The impact mitigation control output limiting unit **52** sets the impact mitigation control output Iref**2**′ by limiting an upper limit of impact mitigation control output Iref**2** output from the impact mitigation control unit **49** by a limiting value 0, Limit**1**, or Limit**2** output from the learning

state determination unit **51** and limiting a lower limit of the impact mitigation control output Iref2 by a limiting value 0, -Limit1, or -Limit2 output from the learning state determination unit **51**. [0051] The impact mitigation control unit **49** mitigates impact and hit sound (abnormal noise) due to end-abutting by suppressing increase in the steering angle θ h when the steering angle θ h comes close to a rack end position. Control performed by the impact mitigation control unit **49** to mitigate impact and abnormal noise due to end-abutting is sometimes referred to as "impact mitigation control".

[0052] The impact mitigation control unit 49 outputs a current command value to suppress increase in the steering angle θ h in order to mitigate impact and hit sound due to end-abutting, as the impact mitigation control output Iref2 at the time of rightward steering has a negative value and reduces the magnitude of the positive basic current command value Iref1. On the other hand, the impact mitigation control output Iref2 at the time of leftward steering has a positive value and reduces the magnitude of the negative basic current command value Iref1. For example, the impact mitigation control unit 49 may output a current command value to generate a steering reaction force.

[0053] The impact mitigation control output limiting unit **52** limits the upper limit of the impact mitigation control output Iref**2** at the time of leftward steering to a limiting value 0 or a positive limiting value Limit**1** or Limit**2** output from the learning state determination unit **51** and limits the lower limit of the impact mitigation control output Iref**2** at the time of rightward steering to a limiting value 0 or a negative limiting value –Limit**1** or –Limit**2** output from the learning state determination unit **51**.

[0054] Details of the impact mitigation control unit **49**, the learning state determination unit **51**, and the relearning determination unit **53** will be described later.

[0055] The current command value Iref3 that the adder 41 calculated is input to the subtracter 42, and deviation Iref3-Im of the current command value Iref3 from a fed-back motor current value Im is calculated. The deviation Iref3-Im is controlled by the current control unit 43 configured to perform PI control or the like, a current-controlled voltage control value Vref is input to the PWM control unit 44 and a duty cycle is calculated therein, and a PWM signal PWM-drives the motor 20 via the inverter 45. The motor current value Im of the motor 20 is detected by the current detector 50, and input to the subtracter 42 and thereby fed back.

[0056] The terminal position learning unit **46** learns, based on the steering angle θ h detected by the steering angle sensor **14**, virtual rack end positions θ evr and θ evl that are terminal positions of a turning mechanism. θ evr denotes a virtual rack end position at the time of rightward steering and has a positive value. θ evl denotes a virtual rack end position at the time of leftward steering and has a negative value.

[0057] Further, error sometimes occurs between a central position between right and left actual rack end positions (hereinafter, sometimes referred to as "rack neutral position") and a neutral position of the steering angle θ h of the column shaft detected by the steering angle sensor **14** (hereinafter, sometimes referred to as "steering angle neutral position"). Hereinafter, such error is sometime referred to as "offset error".

[0058] The offset error occurs due to reasons such as incorrect assembly of the intermediate shaft **4**. The terminal position learning unit **46** estimates offset error Ofs and outputs a corrected steering angle θ h**1** to which the steering angle θ h detected by the steering angle sensor **14** is corrected by subtracting the offset error Ofs from the steering angle θ h. Details of the terminal position learning unit **46** will be described later.

[0059] The control rotational displacement setting unit **47** sets control rotational displacement θ r that indicates how close the corrected steering angle θ h**1** is to one of the virtual rack end positions θ evr and θ evl when the corrected steering angle θ h**1** comes close to a rack end position and is within a range where the impact mitigation control is executed (hereinafter, sometimes referred to as "impact mitigation control execution range").

[0060] FIG. **3** is now referred to. In the case of the rightward steering (that is, in the case where the corrected steering angle $\theta h \mathbf{1}$ is a positive value), a range where the corrected steering angle $\theta h \mathbf{1}$ is greater than a threshold value θthR is the impact mitigation control execution range, and in the case of the leftward steering (that is, in the case where the corrected steering angle $\theta h \mathbf{1}$ is a negative value), a range where the corrected steering angle $\theta h \mathbf{1}$ is less than a threshold value θthL is the impact mitigation control execution range.

[0061] The threshold values θ thR and θ thL are set based on the virtual rack end positions θ evr and θ evl, respectively. For example, the threshold value θ thR for the rightward steering may be a value θ evr- $\Delta\theta$ obtained by subtracting a positive predetermined value $\Delta\theta$ from the virtual rack end position θ evr, and the threshold value θ thL for the leftward steering may be a value θ evl+ $\Delta\theta$ obtained by adding the predetermined value $\Delta\theta$ to the virtual rack end position θ evl. [0062] The control rotational displacement θ r is, for example, set to zero ("0") outside the impact mitigation control execution range (that is, θ thL $\leq \theta$ thR), and, within the impact mitigation control execution range for the rightward steering, the larger a difference θ thP θ thR obtained by subtracting the threshold value θ thR from the corrected steering angle θ th becomes, the larger the control rotational displacement θ r is set. In contrast, within the impact mitigation control execution range for the leftward steering, the smaller a difference θ thP θ thL obtained by subtracting the threshold value θ thL from the corrected steering angle θ th becomes (that is, the larger an absolute value $|\theta$ thP θ thL| becomes), the smaller the negative control rotational displacement θ r may be set (that is, an absolute value $|\theta$ th| becomes larger).

[0063] In other words, in a range where the corrected steering angle $\theta h\mathbf{1}$ is greater than the threshold value θthR , the positive control rotational displacement Or increases in accordance with increase in the corrected steering angle $\theta h\mathbf{1}$, and, in a range where the corrected steering angle $\theta h\mathbf{1}$ is less than the threshold value θthL , the negative control rotational displacement θr decreases in accordance with decrease in the corrected steering angle $\theta h\mathbf{1}$.

[0064] For example, the control rotational displacement setting unit **47** may set a difference $\theta h \mathbf{1} - \theta th R$ as the control rotational displacement θr when the corrected steering angle $\theta h \mathbf{1}$ is greater than the threshold value $\theta th R$, and may set a difference $\theta h \mathbf{1} - \theta th L$ as the control rotational displacement θr when the corrected steering angle $\theta h \mathbf{1}$ is less than the threshold value $\theta th L$. [0065] FIG. **2** is now referred to. The differentiating unit **48** differentiates the steering angle $\theta h t d t$ detected by the steering angle sensor **14** and thereby calculates steering angular velocity $\theta t d t$. [0066] The impact mitigation control unit **49** sets the impact mitigation control output Iref**2**, based on the control rotational displacement $\theta r t d t t$ and the steering angular velocity $\theta t d t t$.

[0067] FIG. **4** is a block diagram illustrative of an example of a functional configuration of the impact mitigation control unit **49**. The impact mitigation control unit **49** includes a spring constant table **60**, multipliers **61** and **63**, a viscosity constant table **62**, an adder **64**, an inverter **65**, and a limiter **66**.

[0068] The spring constant table **60** is a data table calculating a spring constant k0 of a steering system. The spring constant k0 has a characteristic of comparatively steeply increasing (nonlinearly increasing) at a middle portion of a change region as the control rotational displacement θ r increases, as illustrated in FIG. **5A**. Note that a characteristic in the case where the control rotational displacement θ r is a negative value is a characteristic that is line symmetric to the characteristic in FIG. **5A** with the spring-constant-k0 axis (ordinate axis) as the symmetry axis. [0069] The viscosity constant table **62** is a data table calculating a viscosity constant μ of the steering system. The viscosity constant μ has a characteristic of comparatively gently and gradually increasing (nonlinearly increasing) over the entire range as the control rotational displacement θ r increases, as illustrated in FIG. **5B**. Note that a characteristic in the case where the control rotational displacement θ r is a negative value is a characteristic that is line symmetric to the characteristic in FIG. **5B** with the viscosity-constant- μ axis (ordinate axis) as the symmetry axis. [0070] The spring constant k0 from the spring constant table **60** is multiplied by the control

rotational displacement θr by the multiplier $\mathbf{61}$, and a multiplication result $k0 \times \theta r$ is input to the adder $\mathbf{64}$. In addition, the viscosity constant μ from the viscosity constant table $\mathbf{62}$ is multiplied by the steering angular velocity ω by the multiplier $\mathbf{63}$, and a multiplication result $\mu \times \omega$ is input to the adder $\mathbf{64}$. An addition result (= $k0 \times \theta r + \mu \times \omega$) by the adder $\mathbf{64}$ is input to the inverter $\mathbf{65}$ and the limiter $\mathbf{66}$, and the impact mitigation control output Iref2 the sign of which is inverted and the maximum value of which is limited is set.

[0071] Note that the configuration of the impact mitigation control unit **49** in FIG. **4** is only an exemplification and the present invention is not limited to the above-described configuration. The impact mitigation control unit **49** is only required to have a configuration capable of outputting the impact mitigation control output Iref**2** that suppresses increase in the steering angle θh when the corrected steering angle θh comes close to a rack end position.

[0072] Next, details of the terminal position learning unit **46** will be described. The terminal position learning unit **46** calculates, within a range of angle that the steering angle θ h detected by the steering angle sensor **14** when rotational force applied to the turning mechanism is less than or equal to a first predetermined value can take, a steering angle that is positioned farthest from the steering angle neutral position (in the case of the positive steering angle θ h, the maximum steering angle, and in the case of the negative steering angle θ h, the minimum steering angle) as a first candidate θ m**1** of the virtual rack end.

[0073] For example, the terminal position learning unit **46** may calculate a first candidate θ m**1** of the virtual rack end when column output shaft torque Tc applied to the column output shaft **2**o is less than or equal to a predetermined value T**1**.

[0074] The terminal position learning unit **46** may calculate, within a range of angle that the steering angle θ h detected by the steering angle sensor **14** when the rotational force applied to the turning mechanism is less than or equal to the first predetermined value and operational force acting on a steering operation unit is less than or equal to a third predetermined value can take, a steering angle that is positioned farthest from the steering angle neutral position as the first candidate θ m**1** of the virtual rack end.

[0075] For example, the terminal position learning unit **46** may calculate the first candidate $\theta m \mathbf{1}$ when the column output shaft torque Tc is less than or equal to the predetermined value T**1** and the steering torque Th is less than or equal to a predetermined value T**2**.

[0076] In addition, the terminal position learning unit **46** calculates, within a range of an angle obtained by shifting the steering angle θ h detected by the steering angle sensor **14** in a direction toward the steering angle neutral position by a second predetermined value, a steering angle that is positioned farthest from the steering angle neutral position (that is, in the case of the positive steering angle θ h, an angle obtained by subtracting the second predetermined value from the maximum steering angle, and in the case of the negative steering angle θ h, an angle obtained by adding the second predetermined value to the minimum steering angle) as a second candidate θ m**2** of the virtual rack end. As the second predetermined value, for example, a maximum value that can be taken as error may be set.

[0077] The terminal position learning unit **46** selects a steering angle that is positioned farthest from the steering angle neutral position among the first candidate $\theta m\mathbf{1}$ and the second candidate $\theta m\mathbf{2}$ as the virtual rack end positions θevr and θevl .

[0078] This configuration can reduce influence of torsion or the like occurring due to torque and reduce error between the virtual rack end positions θ evr and θ evl and actual rack end positions. [0079] FIG. **6** is a block diagram illustrative of an example of a functional configuration of the terminal position learning unit **46**. The terminal position learning unit **46** includes an output shaft torque calculation unit **70**, a selection unit **71**, a first storage unit **72**, delay units **73** and **77**, a rate limiter **74**, a corrected position calculation unit **75**, a second storage unit **76**, a third storage unit **78**, a limiter **79**, a stroke calculation unit **80**, an offset error calculation unit **81**, a subtracter **82**, and a terminal position correction unit **83**.

[0080] The output shaft torque calculation unit **70** calculates the column output shaft torque Tc applied to the column output shaft **20**.

[0081] For example, the output shaft torque calculation unit **70** may calculate, as the column output shaft torque Tc, motor torque estimated by multiplying the current command value Iref**3** for the motor **20** or the motor current value Im detected by the current detector **50** by a motor torque constant and a reduction ratio of the reduction gear **3**.

[0082] For example, the output shaft torque calculation unit **70** may calculate, as the column output shaft torque Tc, a sum of motor torque estimated by multiplying the current command value Iref**3** for the motor **20** by the motor torque constant and the reduction ratio of the reduction gear **3** and the steering torque Th detected by the torque sensor **10**.

[0083] In addition, for example, the output shaft torque calculation unit **70** may also calculate, as the column output shaft torque Tc, a sum of motor torque estimated by multiplying the motor current value Im detected by the current detector **50** by the motor torque constant and the reduction ratio of the reduction gear **3** and the steering torque Th detected by the torque sensor **10**. [0084] In addition, the output shaft torque calculation unit **70** may calculate motor angular acceleration by second-order differentiating a detected value of the angle sensor of the motor **20**, estimate inertia torque by multiplying the motor angular acceleration by an inertia moment, and add the inertia torque to the column output shaft torque Tc calculated as described above. [0085] The column output shaft torque Tc is an example of the "rotational force applied to the turning mechanism". The steering torque Th is an example of the "operational force acting on a steering operation unit of the vehicle".

[0086] The subtracter **82** calculates the corrected steering angle $\theta h 1$ by subtracting the offset error Ofs calculated by the offset error calculation unit **81** from the steering angle θh detected by the steering angle sensor **14**. The calculation of the offset error Ofs by the offset error calculation unit **81** will be described later.

[0087] The selection unit **71** selects one of the corrected steering angle $\theta h 1$ and output from the delay unit **73** depending on values of the column output shaft torque Tc and the steering torque Th and outputs the selected one to the first storage unit **72**. The delay unit **73** delays the first candidate $\theta m 1$ of the virtual rack end that is stored in and output from the first storage unit **72** and outputs the delayed first candidate $\theta m 1$.

[0088] For example, the selection unit **71** may, when the column output shaft torque Tc is less than or equal to the predetermined value T**1** and the steering torque Th is less than or equal to the predetermined value T**2**, select the corrected steering angle θ h**1** calculated from the detected steering angle θ h and output the corrected steering angle θ h**1** to the first storage unit **72** and, otherwise, output the output from the delay unit **73** to the first storage unit **72**.

[0089] The first storage unit **72** stores the position farther from the steering angle neutral position than the other among the output from the delay unit **73** and the corrected steering angle $\theta h 1$ as the first candidate $\theta m 1$ of the virtual rack end.

[0090] Because of this configuration, when the corrected steering angle $\theta h 1$ that is calculated when the column output shaft torque Tc is less than or equal to the predetermined value T1 and the steering torque Th is less than or equal to the predetermined value T2 is positioned farther from the steering angle neutral position than the first candidate $\theta m 1$ having been stored in the first storage unit 72 up to that time, the first candidate $\theta m 1$ stored in the first storage unit 72 is updated by the corrected steering angle $\theta h 1$.

[0091] Note that the selection unit **71** may, when the column output shaft torque Tc is less than or equal to the predetermined value T**1**, select the corrected steering angle θh **1** and output the corrected steering angle θh **1** to the first storage unit **72** and, otherwise, output the output from the delay unit **73** to the first storage unit **72**.

[0092] The rate limiter **74** accepts as input the first candidate θ m**1** output from the first storage unit **72** and a steering angle do output from the third storage unit. The rate limiter **74** limits a change

rate of the first candidate $\theta m \mathbf{1}$ with respect to the steering angle θo delayed by a delay unit (not illustrated) and outputs a first candidate $\theta m \mathbf{1}'$ generated by limiting the change rate of the first candidate $\theta m \mathbf{1}$ to the third storage unit **78**.

[0093] The corrected position calculation unit **75** calculates an angle obtained by shifting the corrected steering angle $\theta h 1$ in the direction toward the steering angle neutral position by the second predetermined value. That is, when the corrected steering angle $\theta h 1$ is positive, the corrected position calculation unit **75** outputs an angle obtained by subtracting the second predetermined value from the corrected steering angle $\theta h 1$. When the corrected steering angle $\theta h 1$ is negative, the corrected position calculation unit **75** outputs an angle obtained by adding the second predetermined value to the corrected steering angle $\theta h 1$.

[0094] The second storage unit **76** stores the position farther from the steering angle neutral position than the other among the output from the corrected position calculation unit **75** and output from the delay unit **77** as the second candidate θ m**2** of the virtual rack end. The delay unit **77** delays the second candidate θ m**2** of the virtual rack end that is stored in and output from the second storage unit **76** and outputs the delayed second candidate θ m**2**.

[0095] Because of this configuration, when the output from the corrected position calculation unit **75** (that is, an angle obtained by shifting the corrected steering angle $\theta h1$ in the direction toward the steering angle neutral position by the second predetermined value) is positioned farther from the steering angle neutral position than the second candidate $\theta m2$ having been stored in the second storage unit **76** up to that time, the second candidate $\theta m2$ stored in the second storage unit **76** is updated by the output from the corrected position calculation unit **75**. The third storage unit **78** stores the position farther from the steering angle neutral position than the other among the first candidate $\theta m1$ the change rate of which is limited by the rate limiter **74** and the second candidate $\theta m2$ as the steering angle do and outputs the steering angle $\theta 0$.

[0096] The limiter **79** limits the magnitude of the steering angle θ **0**, which is output from the third storage unit **78**, and outputs the limited steering angle do as the virtual rack end positions θ evr and θ evl.

[0097] With reference to FIGS. 7A and 7B, a learning example of a virtual rack end according to the present embodiment will be described. For simplification of description, it is assumed that the offset error Ofs is 0 (that is, (steering angle θh)=(corrected steering angle θh 1) holds), and a case is described where when the steering angle θh that is detected when the column output shaft torque Tc is less than or equal to the predetermined value T1 is positioned farther from the steering angle neutral position than the first candidate θm 1 having been stored in the first storage unit 72 up to that time, the first candidate θm 1 is updated.

[0098] FIG. 7A is an explanatory diagram of an example of change in the column output shaft torque Tc associated with change in the steering angle θ h. Arrows in the drawing indicate steering directions.

[0099] At the time of additional turning of the steering wheel, the column output shaft torque Tc exceeds the predetermined value T1 when the steering angle θ h exceeds θ 1, and at the time of subsequent reverse turning of the steering wheel, the column output shaft torque Tc becomes less than the predetermined value T1 when the steering angle θ h becomes less than θ 2 (θ 2> θ 1). [0100] FIG. 7B is an explanatory diagram of an example of a learned value of a virtual rack end in the rightward steering when the column output shaft torque Tc in FIG. 7A is generated. The dashed line, the dashed-dotted line, the dashed-two-dotted line, and the solid line indicate the steering angle θ h, the first candidate θ m1, the second candidate θ m2, and the output do from the third storage unit 78 (the virtual rack end positions θ evr and θ evl before being limited by the limiter 79), respectively. Note that the dashed-dotted line and the dashed-two-dotted line are illustrated at slightly shifted positions lest the dashed-dotted line and the dashed-two-dotted line overlap other lines.

[0101] When, at time t1, the steering angle θ h increases and the additional turning of the steering

wheel is started, while the column output shaft torque Tc is less than or equal to the predetermined value T1 (that is, while the steering angle θ h is less than or equal to θ 1), the steering angle θ h (dashed line) is learned as the first candidate θ m1 (dashed-dotted line). During the additional turning of the steering wheel, the first candidate θ m1 (dashed-dotted line) increases to θ 1. [0102] In addition, an angle obtained by subtracting the second predetermined value from the steering angle θ h is learned as the second candidate θ m2 (dashed-two-dotted line).

[0103] Thus, while the first candidate $\theta m 1'$ the change rate of which is limited by the rate limiter **74** is greater than the second candidate $\theta m 2$ (dashed-two-dotted line) (from time t 1 to time t 2), the first candidate $\theta m 1'$ is selected as the output θo (solid line) from the third storage unit **78**, and when, at time t 2, the second candidate $\theta m 2$ exceeds the first candidate $\theta m 1'$, the second candidate $\theta m 2$ is selected as the output to (solid line).

[0104] Subsequently, when, at time t3, the steering angle θh ceases to increase and stays at a constant value, the second candidate $\theta m2$ (dashed-two-dotted line) also ceases to increase. Thus, thereafter, the first candidate $\theta m1$ the change rate of which is limited by the rate limiter **74** is selected as the output θo (solid line) from the third storage unit **78**.

[0105] Since, as described above, the first candidate $\theta m1$ (dashed-dotted line) increases to $\theta 1$, the output θo (solid line) from the third storage unit θo (solid line) reaches θo (solid line) reaches θo (solid line) reaches θo (the output θo ceases to increase. [0106] When, subsequently, the steering angle θo starts to decrease and the reverse turning of the steering wheel is started, the steering angle θo decreases to θo at time θo (solid line) reaches θo at time θo (solid line) reaches θo at time θo (solid line) reaches θo at time θo (solid line) reaches θo at time θo (solid line) reaches θo at time θo (solid line) reaches θo at time θo (solid line) reaches θo at time θo (solid line) reaches θo at time θo (solid line) reaches θo at time θo (solid line) reaches θo at time θo a

[0107] Thus, the first candidate $\theta m 1'$ the change rate of which is limited by the rate limiter **74** starts to increase and is selected as the output θo (solid line) from the third storage unit **78**. The output θo (solid line) increases until reaching $\theta 2$ at time t 6 and subsequently becomes constant.

[0108] When comparing the output θ o (solid line) from the third storage unit **78** that is learned as described above, that is, the virtual rack end positions θ evr and θ evl before being limited by the limiter **79**, with a case where an angle obtained by simply subtracting the second predetermined value (for example, a maximum error estimation value) from the steering angle θ h is learned (dashed-two-dotted line), the output θ o from the third storage unit **78** can be learned as a steering angle positioned farther from a steering angle neutral point than the other. Thus, a steering angle positioned closer to an actual rack end position can be learned as the virtual rack end positions θ evr and θ evl.

[0109] Next, the learning state determination unit **51** will be described. FIG. **2** is now referred to. The learning state determination unit **51** determines, based on the virtual rack end positions θ evr and θ evl output from the terminal position learning unit **46**, a state of learning of a virtual rack end position by the terminal position learning unit **46**.

[0110] The learning state determination unit **51** outputs, depending on a determination result on the state of learning of a virtual rack end position, one of 0 and positive limiting values Limit**1** and Limit**2** to the impact mitigation control output limiting unit **52** as a limiting value to limit the upper limit of the impact mitigation control output Iref**2** at the time of leftward steering.

[0111] The limiting value Limit2 is a larger value than the limiting value Limit1, and may, for example, be set to a sufficiently large value to effectively prevent impact and hit sound (abnormal noise) due to end-abutting. On the other hand, the limiting value Limit1 may be set to a value that, although allowing a certain level of impact and hit sound (abnormal noise), can prevent damage to the steering mechanism due to end-abutting.

[0112] The learning state determination unit **51** also outputs, depending on a determination result on the state of learning of a virtual rack end position, one of 0 and negative limiting values –Limit**1** and –Limit**2** to the impact mitigation control output limiting unit **52** as a limiting value to limit the lower limit of the impact mitigation control output Iref**2** at the time of rightward steering.

[0113] Specifically, as initial values before start of learning of the virtual rack end positions θ evr and θ evl, a positive initial value θ int and a negative initial value $-\theta$ int are stored, respectively, in the first storage unit **72**, the second storage unit **76**, and the third storage unit **78**.

[0114] The initial values θ int and $-\theta$ int may be appropriately set in such a way that there is no possibility that the initial values θ int and $-\theta$ int are positioned on the outer side of actual rack end positions (that is, in such a way that there is no possibility that the initial values θ int and $-\theta$ int are positioned farther from the steering angle neutral point than the actual rack end positions). For example, the initial values θ int and $-\theta$ int may be set in such a manner that θ int=(rack stroke minimum value Stmin)-(rack end maximum value fevmax) holds.

[0115] In the setting, the rack stroke minimum value Stmin may be set to a minimum value of variation in values that can be calculated as a rack stroke between the virtual rack end positions θ evr and θ evl (for example, a lower limit of manufacturing tolerance).

[0116] In addition, the "rack end maximum value θ evmax" is a maximum value of absolute values of values that can be learned as the virtual rack end positions θ evr and θ evl and may be set in such a manner that the rack end maximum value θ evmax=(rack stroke maximum value Stmax)/2+ (estimated value of offset error between rack neutral position and steering angle neutral position) holds.

[0117] In addition, the rack stroke maximum value Stmax is a maximum value of variation in values that can be calculated as the rack stroke between the virtual rack end positions θ evr and θ evl and may, for example, be set to a value obtained by adding learning error of the virtual rack end positions θ evr and θ evl to an upper limit of manufacturing tolerance.

[0118] When the right virtual rack end position θ evr output from the terminal position learning unit **46** is less than a predetermined learning threshold value θ lth, the learning state determination unit **51** determines that learning of the right virtual rack end position θ evr has not been performed and outputs "0" as a limiting value to limit the lower limit of the impact mitigation control output Iref2. [0119] As the "learning threshold value θ lth", a minimum value of the absolute values of values that can be learned as the virtual rack end positions θ evr and θ evl may be set, and the "learning threshold value θ lth" may, for example, be set in such a manner that the learning threshold value θ lth=(rack stroke minimum value Stmin)/2–(estimated value of offset error between rack neutral position and steering angle neutral position) holds.

[0120] Likewise, when the left virtual rack end position θ evl is greater than a negative learning threshold value $-\theta$ lth (that is, the absolute value $|\theta$ evl| is less than the absolute value $|\theta$ lth|), the learning state determination unit **51** determines that learning of the left virtual rack end position θ evl has not been performed and outputs "0" as a limiting value to limit the upper limit of the impact mitigation control output Iref**2**.

[0121] When the right virtual rack end position θ evr is greater than or equal to the predetermined learning threshold value θ lth, the learning state determination unit $\mathbf{51}$ determines that the learning of the right virtual rack end position θ evr has been performed and outputs "-Limit1" as a limiting value to limit the lower limit of the impact mitigation control output Iref2.

[0122] Likewise, when the left virtual rack end position θ evl is less than or equal to the negative learning threshold value $-\theta$ lth (that is, the absolute value $|\theta$ evl| is greater than or equal to the absolute value $|\theta$ lth|), the learning state determination unit **51** determines that the learning of the left virtual rack end position θ evl has been performed and outputs "Limit**1**" as a limiting value to limit the upper limit of the impact mitigation control output Iref**2**.

[0123] Further, the learning state determination unit **51** calculates distance between the right virtual rack end position θ evr and the left virtual rack end position θ evl as rack stroke St.

[0124] When the absolute values of learned values of the virtual rack end positions θ evr and θ evl become large and the rack stroke St becomes longer than the rack stroke minimum value Stmin, the learning state determination unit **51** determines that the learning of the virtual rack end positions θ evr and θ evl has been completed and outputs "–Limit2" and "Limit2" as a limiting value to limit

- the lower limit of the impact mitigation control output Iref2 and a limiting value to limit the upper limit of the impact mitigation control output Iref2, respectively.
- [0125] Next, an example of a manner in which, depending on a change in the state of learning of the virtual rack end positions, the limiting value that the learning state determination unit **51** outputs changes will be described.
- [0126] FIGS. **8**A, **8**B, **8**C, **8**D, **8**E, and **8**F are a conceptual diagram of actual rack end positions, a conceptual diagram of a state before start of learning of virtual rack end positions, a conceptual diagram of a state in which the right virtual rack end position θ evr has been learned, a conceptual diagram of a state in which the left virtual rack end position θ evl has been learned, a conceptual diagram of a state in which the learning of the virtual rack ends θ evr and θ evl is considered to have been completed, and a conceptual diagram of a state in which the learning of the virtual rack ends has been continued to the vicinities of the actual rack ends, respectively.
- [0127] In addition, limiting values in a table in FIG. **10** indicate limiting values that the learning state determination unit **51** outputs in the states illustrated in FIGS. **8**B to **8**F.
- [0128] In FIGS. **8**B to **8**F, "0 [deg]" indicates the steering angle neutral position. The same applies to FIGS. **9**B to **9**H, which are described later.
- [0129] In FIGS. **8**B to **8**F, the steering angle neutral position substantially coincides with the rack neutral position (the center position between the actual rack end positions).
- [0130] In the state before start of learning of the virtual rack end positions (FIG. **8**B), the right virtual rack end position θ evr, which is output from the terminal position learning unit **46**, is θ int with reference to the steering angle neutral position and less than the learning threshold value θ lth. [0131] Therefore, the learning state determination unit **51** determines that the learning of the right virtual rack end position θ evr has not been performed and outputs "0" as a limiting value to limit the lower limit of the impact mitigation control output Iref**2** (see FIG. **10**).
- [0132] In addition, the left virtual rack end position θevl , which is output from the terminal position learning unit 46, is $-\theta$ int with reference to the steering angle neutral position and greater than the learning threshold value $-\theta$ lth. Therefore, the learning state determination unit 51 determines that the learning of the left virtual rack end position θevl has not been performed and outputs "0" as a limiting value to limit the upper limit of the impact mitigation control output Iref2 (see FIG. 10). [0133] Subsequently, as illustrated in FIG. 8C, the right virtual rack end position θevr is learned. Since the right virtual rack end position θevr is greater than or equal to the learning threshold value θ lth, the learning state determination unit 51 determines that the learning of the right virtual rack end position θevr has been performed and outputs "–Limit1" as a limiting value to limit the lower limit of the impact mitigation control output Iref2. On the other hand, since the left virtual rack end position θevl has not changed, the learning state determination unit 51 outputs "0" as a limiting value to limit the upper limit of the impact mitigation control output Iref2 (see FIG. 10). [0134] Subsequently, as illustrated in FIG. 8D, the left virtual rack end position θevl is learned. Since the left virtual rack end position θevl becomes less than or equal to the negative learning
- threshold value $-\theta$ lth, the learning state determination unit **51** determines that the learning of the left virtual rack end position θevl has been performed. However, since the rack stroke St is less than or equal to the rack stroke minimum value Stmin, the learning state determination unit **51** does not determine that the learning of the virtual rack end positions θevr and θevl has been completed. [0135] Therefore, the learning state determination unit **51** outputs "Limit**1**" as a limiting value to limit the upper limit of the impact mitigation control output Iref**2**. The learning state determination unit **51** also outputs "–Limit**1**" as a limiting value to limit the lower limit of the impact mitigation control output Iref**2**.
- [0136] Subsequently, as illustrated in FIG. **8**E, further learning of the right virtual rack end position θ evr causes the rack stroke St to become longer than the rack stroke minimum value Stmin. Therefore, the learning state determination unit **51** determines that the learning of the virtual rack end positions θ evr and θ evl has been completed and outputs "Limit**2**" and "–Limit**2**" as limiting

- values to limit the upper limit and the lower limit of the impact mitigation control output Iref2, respectively.
- [0137] Subsequently, repeating the learning of the virtual rack end positions θ evr and θ evl causes the virtual rack end positions θ evr and θ evl to come close to the actual rack end positions (see FIG. **8**F).
- [0138] The learning state determination unit **51** outputs "Limit**2**" and "–Limit**2**" as limiting values to limit the upper limit and the lower limit of the impact mitigation control output Iref**2**, respectively, until the rack stroke St exceeds the rack stroke maximum value Stmax.
- [0139] Next, operation in the case where offset error occurs between the rack neutral position and the steering angle neutral position will be described. FIG. **9**A is a conceptual diagram of the actual rack end positions, FIG. **9**B is the same drawing as FIG. **8**F and illustrates a state in which there is no offset error, and FIG. **9**C is a conceptual diagram of a state immediately after offset error has occurred.
- [0140] The steering angle neutral position in FIG. **9**B substantially coincides with the rack neutral position (the center position between the actual rack end positions), and the steering angle neutral position in FIG. **9**C is shifted to the right side from the rack neutral position by $\Delta\theta$. That is, when the steering wheel is steered to the left side of the steering angle neutral position by $\Delta\theta$, the rack **5***b* is positioned at the rack neutral position.
- [0141] When offset error occurs, the impact mitigation control cannot be normally performed. In the example in FIG. **9**C, since the right virtual rack end position θ evr is positioned on the outer side of the actual rack end position, necessary reduction in impact and abnormal noise cannot be performed.
- [0142] Thus, the terminal position learning unit **46**, as described above, estimates an offset error Ofs between the rack neutral position and the steering angle neutral position and outputs the corrected steering angle $\theta h \mathbf{1}$, to which the steering angle θh detected by the steering angle sensor **14** is corrected by subtracting the offset error Ofs from the steering angle θh .
- [0143] FIG. **6** is now referred to. The stroke calculation unit **80** calculates the rack stroke St. The offset error calculation unit **81** compares the rack stroke St with the rack stroke maximum value Stmax.
- [0144] When the rack stroke St exceeds the rack stroke maximum value Stmax, the offset error calculation unit **81** determines that offset error has occurred. The offset error calculation unit **81** calculates a difference obtained by subtracting the rack stroke maximum value Stmax from the rack stroke St as the offset error Ofs=(rack stroke St)–(rack stroke maximum value Stmax).
- [0145] FIG. **9**D is a conceptual diagram of a state in which, after offset error has occurred, the left virtual rack end position θ evl is learned.
- [0146] Since the steering angle neutral position is shifted to the right side of the rack neutral position, when the left virtual rack end position θ evl is newly learned, the rack stroke St between the virtual rack end positions θ evr and θ evl exceeds the rack stroke maximum value Stmax. The offset error calculation unit **81** calculates a difference obtained by subtracting the rack stroke maximum value Stmax from the rack stroke St (rack stroke St-rack stroke maximum value Stmax) as the offset error Ofs.
- [0147] In the following description, a virtual rack end position that is one of the right and left virtual rack end positions that is learned when a rack stroke St exceeding the rack stroke maximum value Stmax is calculated is sometimes referred to as "one virtual rack end position". In addition, a virtual rack end position that is one of the right and left virtual rack end positions that is not the one virtual rack end position is sometimes referred to as "the other virtual rack end position".
- [0148] When, as in the example in FIG. **9**C, the steering angle neutral position is shifted to the right side from the rack neutral position, the left virtual rack end position serves as one virtual rack end position and the right virtual rack end position serves as the other virtual rack end position. Conversely, when the steering angle neutral position is shifted to the left side from the rack neutral

position, the right virtual rack end position serves as one virtual rack end position and the left virtual rack end position serves as the other virtual rack end position.

[0149] FIG. **6** is now referred to. The subtracter **82** calculates the corrected steering angle $\theta h\mathbf{1}$ by subtracting the offset error Ofs calculated by the offset error calculation unit **81** from the steering angle θh detected by the steering angle sensor **14**.

[0150] The terminal position correction unit **83** corrects the first candidate θ m**1** that the first storage unit **72** stores, the second candidate θ m**2** that the second storage unit **76** stores, and the steering angle do that the third storage unit **78** stores according to the offset error Ofs. With reference to FIG. **9**E, the correction processing of the candidates and the steering angle will be described. [0151] The subtracter **82** subtracting the offset error Ofs from the steering angle θ h causes the steering angle neutral position (position at "0 [deg]") to move, as illustrated in FIG. **9**E. [0152] On the other hand, since the left virtual rack end position θ evl (that is, one virtual rack end position) that was learned in FIG. **9**D is a learned value before the steering angle neutral position, which serves as the base point, moves, when the steering angle neutral position is moved as illustrated in FIG. **9**E, it is necessary to correct the left virtual rack end position θ evl in association with this movement.

[0153] The terminal position correction unit **83** corrects the first candidate θ m**1** of the left virtual rack end position stored in the first storage unit **72** with the offset error Ofs. Since the left virtual rack end position is a negative value, the terminal position correction unit **83** corrects the left virtual rack end position by adding the offset error Ofs. The terminal position correction unit **83** also likewise corrects the second candidate θ m**2** and the steering angle do that are stored in the second storage unit **76** and the third storage unit **78**, respectively.

[0154] When the one virtual rack end position is the right virtual rack end position (that is, when the virtual rack end position is a positive value), the terminal position correction unit **83** corrects the right virtual rack end position by subtracting the offset error Ofs.

[0155] In addition, the terminal position correction unit **83** corrects (resets) the other virtual rack end position (in the example in FIG. **9**E, the right virtual rack end position θ evr) in such a way that the rack stroke St between the virtual rack end positions θ evr and θ evl coincides with the predetermined rack stroke minimum value Stmin. This configuration can correct the other virtual rack end position to a position on the inner side of the actual rack end position.

[0156] FIG. **9**F is now referred to. When a new left virtual rack end position θ evl is further learned, the offset error calculation unit **81** calculates a change amount $\Delta\theta$ evl of the virtual rack end position θ evl between before and after update.

[0157] The offset error calculation unit **81**, by adding the change amount $\Delta\theta$ evl to the offset error Ofs before the new left virtual rack end position θ evl is learned, updates the offset error Ofs. This update causes the steering angle neutral position to further move by the change amount $\Delta\theta$ evl. [0158] The terminal position correction unit **83** corrects the first candidate θ m**1** of the one virtual rack end position (the left virtual rack end position) that is stored in the first storage unit **72** with the change amount $\Delta\theta$ evl. Since the left virtual rack end position is a negative value, the terminal position correction unit **83** corrects the left virtual rack end position by adding the change amount $\Delta\theta$ evl. The terminal position correction unit **83** also likewise corrects the second candidate θ m**2** and the steering angle do that are stored in the second storage unit **76** and the third storage unit **78**, respectively.

[0159] When the one virtual rack end position is the right virtual rack end position (that is, the virtual rack end position is a positive value), the terminal position correction unit **83** corrects the first candidate $\theta m 1$ by subtracting the change amount $\Delta \theta evl$.

[0160] In addition, the terminal position correction unit **83** corrects (resets) the other virtual rack end position (in the example in FIG. **9**F, the right virtual rack end position θ evr) in such a way that the rack stroke St between the virtual rack end positions θ evr and θ evl coincides with the rack stroke minimum value Stmin.

- [0161] FIG. **9**G is now referred to. When a new right virtual rack end position θ evr (that is, the other virtual rack end position) is learned, the offset error calculation unit **81** does not update the offset error Ofs. That is, the steering angle neutral position is not moved.
- [0162] In addition, the terminal position correction unit $\bf 83$ also does not correct the first candidate $\theta m \bf 1$ that the first storage unit $\bf 72$ stores, the second candidate $\theta m \bf 2$ that the second storage unit $\bf 76$ stores, and the steering angle do that the third storage unit $\bf 78$ stores. Because of this configuration, only the right virtual rack end position θevr is updated in such a manner as to move away from the steering angle neutral position.
- [0163] FIG. **9**H is now referred to. After a new right virtual rack end position θ evr has been learned in FIG. **9**G, even when a new left virtual rack end position θ evl (that is, the one virtual rack end position) is further learned, the terminal position correction unit **83** does not correct (reset) the right virtual rack end position θ evr in such a way that the rack stroke St coincides with the predetermined rack stroke minimum value Stmin.
- [0164] As with FIG. **9**F, the offset error calculation unit **81** updates the offset error Ofs (that is, the steering angle neutral position is modified), which causes the steering angle neutral position to move, and the terminal position correction unit **83** corrects the left virtual rack end position θ evl. On this occasion, the right virtual rack end position θ evr is corrected by adding the offset error Ofs to the virtual rack end position θ evr.
- [0165] In addition, when a new right virtual rack end position θ evr (that is, the other virtual rack end position) is further learned, the steering angle neutral position and the left virtual rack end position θ evl (that is, the one virtual rack end position) are not changed and only the right virtual rack end position θ evr changes.
- [0166] Next, operation of the learning state determination unit $\bf 51$ in the case where offset error occurs will be described. The learning state determination unit $\bf 51$ determines that offset error has occurred when the rack stroke St, which is calculated from the right virtual rack end position θ evr and the left virtual rack end position θ evl, is longer than the rack stroke maximum value Stmax. [0167] When determining that offset error has occurred, the learning state determination unit $\bf 51$ resets each of the limiting values to limit the upper limit and the lower limit of the impact mitigation control output Iref $\bf 2$ to "0"
- [0168] Subsequently, when the right virtual rack end position θ evr is, as with the above description, greater than or equal to the predetermined learning threshold value θ lth, the learning state determination unit $\mathbf{51}$ outputs "-Limit $\mathbf{1}$ " as a limiting value to limit the lower limit of the impact mitigation control output Iref $\mathbf{2}$. When the left virtual rack end position θ evl is less than or equal to the negative learning threshold value $-\theta$ lth, the learning state determination unit $\mathbf{51}$ outputs "Limit $\mathbf{1}$ " as a limiting value to limit the upper limit of the impact mitigation control output Iref $\mathbf{2}$. When the rack stroke St becomes longer than the rack stroke minimum value Stmin, the learning state determination unit $\mathbf{51}$ outputs "Limit $\mathbf{2}$ " and "-Limit $\mathbf{2}$ " as limiting values to limit the upper limit and the lower limit of the impact mitigation control output Iref $\mathbf{2}$, respectively.
- [0169] With reference to FIGS. **9**A to **9**F and **10**, an example of limiting values that the learning state determination unit **51** outputs when offset error has occurred will be described. Limiting values in the table in FIG. **10** indicate limiting values that the learning state determination unit **51** outputs in the states illustrated in FIGS. **9**B to **9**H.
- [0170] FIG. **9**B is the same as FIG. **8**F and illustrates a state before offset error occurs. The learning state determination unit **51** outputs "Limit**2**" and "–Limit**2**" as limiting values to limit the upper limit and the lower limit of the impact mitigation control output Iref**2**, respectively.
- [0171] Subsequently, when offset error occurs due to a reason such as incorrect assembly of the intermediate shaft **4**, the state in FIG. **9**B transitions to the state illustrated in FIG. **9**C. Since, in this stage, learning of new virtual rack ends θ evr and θ evl has not been performed, the value of the rack stroke St that the learning state determination unit **51** calculates stays at the same value as that in the state in FIG. **9**B. Therefore, the learning state determination unit **51** has not determined that

offset error occurred and has not reset the limiting values to "0". Thus, the learning state determination unit **51** outputs "Limit**2**" and "-Limit**2**".

[0172] When, in FIG. **9**D, a new left virtual rack end position θ evl is learned, the rack stroke St becomes longer than the rack stroke maximum value Stmax. Thus, the learning state determination unit **51** resets the limiting values to limit the upper limit and the lower limit of the impact mitigation control output Iref**2** to "0"

[0173] In addition, since the left virtual rack end position θ evl is less than or equal to the negative learning threshold value $-\theta$ lth as in FIG. **9**E, the learning state determination unit **51** determines that the learning of the left virtual rack end position θevl has been performed and outputs "Limit1" as a limiting value to limit the upper limit of the impact mitigation control output Iref2. [0174] On the other hand, since the right virtual rack end position θ evr is corrected (reset) in such a way that the rack stroke St between the virtual rack end positions θ evr and θ evl coincides with the rack stroke minimum value Stmin, the rack stroke St does not become longer than the rack stroke minimum value Stmin. Therefore, it is not determined that the learning of the virtual rack end positions θ evr and θ evl has been completed, and since the right virtual rack end position θ evr is less than the learning threshold value θ lth, output of "0" as a limiting value to limit the lower limit of the impact mitigation control output Iref2 is maintained. The same applies to the state in FIG. 9F. [0175] FIG. **9**G is now referred to. When a new right virtual rack end position θ evr (that is, the other virtual rack end position) is learned and the rack stroke St becomes longer than the rack stroke minimum value Stmin, the learning state determination unit 51 determines that the learning of the virtual rack end positions θ evr and θ evl has been completed and outputs "Limit2" and "-Limit**2**" as limiting values to limit the upper limit and the lower limit of the impact mitigation control output Iref2, respectively.

[0176] In FIG. **9**H, the learning state determination unit **51** likewise outputs "Limit**2**" and "–Limit**2**" as limiting values to limit the upper limit and the lower limit of the impact mitigation control output Iref**2**, respectively.

[0177] As described above, the turning control device of the present embodiment is capable of limiting the impact mitigation control output Iref2 in a stepwise manner depending on a degree of learning of the virtual rack end positions, based on a comparison result between a learned virtual rack end position and the learning threshold value θ lth and a comparison result between the rack stroke St calculated from the learned virtual rack end position and the rack stroke minimum value Stmin. Because of this configuration, it is possible to learn virtual rack end positions while preventing damage to the steering mechanism due to end-abutting.

[0178] Next, the relearning determination unit 53 will be described. FIG. 2 is now referred to. The relearning determination unit 53 determines necessity of relearning of the virtual rack end position θ evr or θ evl depending on whether or not occurrence of end-abutting is detected. In the following description, the virtual rack end positions θ evr and θ evl are sometimes collectively referred to as "virtual rack end positions θ ev".

[0179] When the relearning of the virtual rack end positions θev is to be performed, the relearning determination unit $\mathbf{53}$ outputs a relearning command signal Cmd commanding the terminal position learning unit $\mathbf{46}$ to perform relearning. Because of this configuration, the virtual rack end positions θev can be relearned when, for example, in association with a rack shaft being replaced after the virtual rack end positions θev are learned, end-abutting occurs due to a reason, such as a rack shaft that has a length different from length of a rack shaft that should be originally mounted being mistakenly mounted or a mounting position of a rack shaft having changed between before and after replacement of the rack shaft.

[0180] FIG. **11** is a block diagram illustrative of an example of a functional configuration of the relearning determination unit **53**. The relearning determination unit **53** includes an end-abutting detection unit **90**, an end-abutting steering angle range determination unit **91**, a counting unit **92**, and a relearning unit **93**.

[0181] The end-abutting detection unit **90** detects occurrence of end-abutting, based on the column output shaft torque Tc and the steering torque Th and retains an end-abutting steering angle θ abt that is a steering angle when occurrence of end-abutting is detected.

[0182] For example, the end-abutting detection unit **90** retains a steering angle θ dt when the steering torque Th is greater than or equal to a predetermined threshold value Tth**1** (for example, 8 Nm) and, based on the retained steering angle θ dt, determines necessity of detecting the end-abutting steering angle θ abt. For example, the end-abutting detection unit **90** may determine that detection of the end-abutting steering angle θ abt is necessary when an absolute value $|\theta ev - \theta dt|$ of a difference between the steering angle θ dt and one of the virtual rack end positions θ ev is greater than a steering angle threshold value θ tha**1** and determine that the detection of the end-abutting steering angle θ abt is not necessary when the absolute value $|\theta ev - \theta dt|$ of the difference between the steering angle θ dt and the virtual rack end position θ ev is less than or equal to the steering angle threshold value θ tha**1**.

[0183] FIG. **12**A is now referred to. FIG. **12**A illustrates as an example a case where the steering wheel **1** is steered to the right. In processing when the steering wheel **1** is steered to the left, the right virtual rack end position θ evr in the following description is replaced by the left virtual rack end position θ evl.

[0184] When the steering angle when the steering torque Th is greater than or equal to the predetermined threshold value Tth1 is a steering angle θ dt1 illustrated in the drawing, the endabutting detection unit **90** determines that the detection of the end-abutting steering angle θ abt is not necessary since the absolute value $|\theta \text{evr} - \theta \text{dt} \mathbf{1}|$ of the difference between the virtual rack end position θ evr and the steering angle θ dt1 is less than or equal to the steering angle threshold value ethal. On the other hand, when the steering angle when the steering torque Th is greater than or equal to the predetermined threshold value Tth1 is a steering angle θ dt2 illustrated in the drawing, the end-abutting detection unit **90** determines that the detection of the end-abutting steering angle θ abt is necessary since the absolute value $|\theta$ evr $-\theta$ dt2| of the difference between the virtual rack end position θ evr and the steering angle θ dt2 is greater than the steering angle threshold value θ tha1. [0185] When determining that the detection of the end-abutting steering angle θ abt is necessary, the end-abutting detection unit **90** retains a steering angle θcd when, after the steering torque Th has become greater than or equal to the predetermined threshold value Tth1, the column output shaft torque Tc becomes less than a predetermined threshold value Tth2 (for example, 35 Nm) and the steering torque Th becomes less than the predetermined threshold value Tth1, as a candidate of the end-abutting steering angle θabt (hereinafter, sometimes referred to as "end-abutting steering angle candidate").

[0186] The end-abutting detection unit $\bf 90$ acquires an end-abutting steering angle candidate θ cd as the end-abutting steering angle θ abt when an absolute value $|\theta ev - \theta cd|$ of a difference between the end-abutting steering angle candidate θ cd and one of the virtual rack end positions θ ev is greater than a steering angle threshold value θ tha2. When the absolute value $|\theta ev - \theta cd|$ is less than or equal to the steering angle threshold value θ tha2, the end-abutting detection unit $\bf 90$ discards the end-abutting steering angle candidate θ cd without acquiring the end-abutting steering angle candidate θ cd as the end-abutting steering angle θ abt. For example, the steering angle threshold value θ tha2 may be set to a value a predetermined margin Δ smaller than the steering angle threshold value θ tha1. For example, the predetermined margin Δ may be set according to detection error of the steering angle θ h.

[0187] FIG. **12**B is now referred to. FIG. **12**B illustrates as an example the case where the steering wheel **1** is steered to the right. In the processing when the steering wheel **1** is steered to the left, the right virtual rack end position θ evr in the following description is replaced by the left virtual rack end position θ evl.

[0188] In a case of an end-abutting steering angle candidate $\theta cd\mathbf{1}$, the end-abutting detection unit **90** discards the end-abutting steering angle candidate $\theta cd\mathbf{1}$ without acquiring the end-abutting

steering angle candidate $\theta cd1$ as the end-abutting steering angle θ abt since an absolute value $|\theta evr-\theta cd1|$ of a difference between the virtual rack end position θevr and the end-abutting steering angle candidate $\theta cd1$ is less than or equal to the steering angle threshold value $\theta tha2$. On the other hand, in a case of an end-abutting steering angle candidate $\theta cd2$, the end-abutting detection unit θd 0 acquires the end-abutting steering angle candidate $\theta cd2$ as the end-abutting steering angle θd 0 at the end-abutting steering angle θd 1 of a difference between the virtual rack end position θd 2 and the end-abutting steering angle candidate $\theta cd2$ is greater than the steering angle threshold value θd 1.

[0189] Next, the end-abutting detection unit **90** calculates a rack stroke approximate value Sta, based on the acquired end-abutting steering angle θ abt. When the rack stroke approximate value Sta is not a value within an allowable range, the end-abutting detection unit **90** may exclude the acquired end-abutting steering angle θ abt from targets of determination processing of variation in end-abutting steering angles θ abt performed by the end-abutting steering angle range determination unit **91**, which will be described later, and discard the acquired end-abutting steering angle θ abt. [0190] FIG. **13**A illustrates a calculation example of the rack stroke approximate value Sta when in a case where the steering wheel 1 is steered to one of the left side and the right side, end-abutting does not occur and in a case where the steering wheel 1 is steered to the other of the left side and the right side, end-abutting occurs and the end-abutting steering angle θ abt is acquired. FIG. **13**B illustrates a calculation example of the rack stroke approximate value Sta when both in the case where the steering wheel **1** is steered to the left side and in the case where the steering wheel **1** is steered to the right side, end-abutting occurs and an end-abutting steering angle θ abtl and an endabutting steering angle θ abtr are acquired both on the left side and on the right side, respectively. [0191] When, as illustrated in FIG. **13**A, the end-abutting steering angle θ abt is acquired when the steering wheel **1** is steered to the right side, the end-abutting detection unit **90** calculates an absolute value of a difference between the left virtual rack end position θ evl and the end-abutting steering angle θ abt on the right side as the rack stroke approximate value Sta.

[0192] When error between the rack stroke approximate value Sta and a predetermined value does not fall within a threshold value, it is considered that the vehicle is in a state in which one of the steered wheels **8**L and **8**R collides with a curb or the like when the steering wheel **1** is steered and the steering wheel **1** cannot be steered to a rack end. Thus, when error between the rack stroke approximate value Sta and the predetermined value does not fall within the threshold value, the end-abutting detection unit **90** may discard the steering angle θ abt without using the steering angle θ abt for determination of necessity of relearning.

[0193] Note that in the processing in a case where the end-abutting steering angle θ abt is acquired when the steering wheel **1** is steered to the left, the left virtual rack end position θ evl in the above description is replaced by the right virtual rack end position θ evr.

[0194] In the following description, an end-abutting steering angle θ abt acquired when the steering wheel **1** is steered to the left side is sometimes referred to as "left end-abutting steering angle", and an end-abutting steering angle θ abt acquired when the steering wheel **1** is steered to the right side is sometimes referred to as "right end-abutting steering angle".

[0195] When, as illustrated in FIG. **13**B, both a left end-abutting steering angle θ abtl and a right end-abutting steering angle θ abtr are acquired, the end-abutting detection unit **90** calculates an absolute value of a difference between the end-abutting steering angles θ abtl and θ abtr as the rack stroke approximate value Sta. The end-abutting detection unit **90** compares a minimum value of a rack stroke of a rack shaft that has a possibility of being mistakenly mounted on the vehicle (sometimes refer to as "erroneous stroke minimum value Sterr") with the rack stroke approximate value Sta. When the rack stroke approximate value Sta is greater than or equal to the erroneous stroke minimum value Sterr, the end-abutting detection unit **90** determines that the end-abutting steering angles θ abtl and θ abtr are valid. When the rack stroke approximate value Sta is less than the erroneous stroke minimum value Sterr, the end-abutting detection unit **90** determines that the

detection of the end-abutting steering angle θ abt is false detection. The end-abutting detection unit **90** outputs a determination result to the counting unit **92**.

[0196] FIG. **11** is now referred to. The end-abutting steering angle range determination unit **91** determines whether or not variation in right end-abutting steering angles θ abt that the end-abutting detection unit **90** acquired multiple times is less than or equal to a predetermined threshold value θ tha3. Likewise, the end-abutting steering angle range determination unit **91** determines whether or not variation in left end-abutting steering angles θ abt that the end-abutting detection unit **90** acquired multiple times is less than or equal to the predetermined threshold value θ tha3. [0197] Although a method for calculating variation in right end-abutting steering angles θ abt will be described below with reference to FIGS. **14**A to **14**C, the same applies to a method for calculating variation in left end-abutting steering angles θ abt.

[0198] FIG. **14**A is now referred to. In FIG. **14**A, a triangular mark indicates an end-abutting steering angle θ abt that the end-abutting detection unit **90** acquired most recently. Hereinafter, an end-abutting steering angle θ abt that the end-abutting detection unit **90** acquired most recently is sometimes referred to as "latest end-abutting steering angle". Circular marks indicate end-abutting steering angles θ abt that the end-abutting detection unit **90** acquired at time points before the endabutting detection unit **90** acquired the latest end-abutting steering angle θ abt (the triangular mark) and that are stored in an end-abutting steering angle storage unit **94**. Hereinafter, an end-abutting steering angle θ abt that is stored in the end-abutting steering angle storage unit **94** is sometimes referred to as "stored end-abutting steering angle". The same applies to FIGS. **14**B and **14**C. [0199] As illustrated in FIG. **14**A, when a difference between a minimum value θ min and a maximum value θ max of the latest end-abutting steering angle θ abt (the triangular mark) and the stored end-abutting steering angles θabt (the circular marks) is less than or equal to the predetermined threshold value θ tha3, the end-abutting steering angle range determination unit 91 determines that variation in the end-abutting steering angles θ abt is less than or equal to the predetermined threshold value θ tha3. In this case, the end-abutting steering angle range determination unit 91 stores the latest end-abutting steering angle θ abt (the triangular mark) in the end-abutting steering angle storage unit **94** in addition to the stored end-abutting steering angles θ abt (the circular marks).

[0200] FIG. **11** is now referred to. The counting unit **92** counts the number of times that the endabutting steering angle range determination unit **92** determines that variation in end-abutting steering angles θ abt is less than or equal to the predetermined threshold value θ tha**3**, every time the end-abutting detection unit **90** acquires an end-abutting steering angle θ abt (a triangular mark). That is, the counting unit **92** counts the number of times of acquisition of an end-abutting steering angle that causes variation in end-abutting steering angles to be less than or equal to the predetermined threshold value θ tha**3**.

[0201] The counting unit **92** includes a vehicle state determination unit **95**, end-abutting counters **96**oL, **96**oR, **96**bL, and **96**bR, provisional end-abutting counters **97**oL, **97**oR, **97**bL, and **97**bR, and on-center counters **98**oL, **98**oR, **98**bL, and **98**bR.

[0202] In the following description, the end-abutting counters **96**oL, **96**oR, **96**bL, and **96**bR are collectively referred to as "end-abutting counter **96**", the provisional end-abutting counters **97**oL, **97**oR, **97**bL, and **97**bR are collectively referred to as "provisional end-abutting counter **97**", and the on-center counters **98**oL, **98**oR, **98**bL, and **98**bR are collectively referred to as "on-center counter **98**".

[0203] The vehicle state determination unit **95** determines whether the vehicle is in a stop state or a travel state, based on the vehicle speed Vh detected by the vehicle speed sensor **12**. FIG. **15** is an example of a state transition diagram of a vehicle state. When the vehicle speed Vh becomes greater than or equal to a vehicle speed threshold value Vth when the vehicle is in the stop state, the vehicle state determination unit **95** determines that the vehicle state has transitioned to the travel state. When the vehicle speed Vh becomes less than the vehicle speed threshold value Vth when the

vehicle is in the travel state and a count value in the end-abutting counter **96** is "0", the vehicle state determination unit **95** determines that the vehicle state has transitioned to the stop state. Since the counting unit **92** includes four end-abutting counters **96**oL, **96**oR, **96**bL, and **96**bR as described above, the vehicle state determination unit **95** may determine whether or not the vehicle state has transitioned to the stop state with respect to each of the counters.

[0204] FIG. **11** is now referred to. The end-abutting counter **96** counts and stores the number of times that the end-abutting steering angle range determination unit **91** determines that variation in end-abutting steering angles θ abt is less than or equal to the predetermined threshold value θ tha**3**, every time the end-abutting detection unit **90** acquires an end-abutting steering angle θ abt (a triangular mark) when the vehicle state is the travel state.

[0205] That is, when the end-abutting steering angle range determination unit **91** determines that a difference between a minimum value θ min and a maximum value θ max of a latest end-abutting steering angle θ abt (a triangular mark) and stored end-abutting steering angles θ abt (circular marks) is less than or equal to the predetermined threshold value θ tha when the vehicle state is the travel state, the end-abutting counter **96** increments a count value by one.

[0206] On the other hand, the provisional end-abutting counter **97** counts the number of times that the end-abutting steering angle range determination unit **91** determines that variation in end-abutting steering angles θ abt is less than or equal to the predetermined threshold value θ tha3, every time the end-abutting detection unit **90** acquires an end-abutting steering angle θ abt (a triangular mark) when the vehicle state is the stop state and stores a counted number.

[0207] That is, when the end-abutting steering angle range determination unit **91** determines that a difference between a minimum value θ min and a maximum value θ max of a latest end-abutting steering angle θ abt (a triangular mark) and stored end-abutting steering angles θ abt (circular marks) is less than or equal to the predetermined threshold value θ tha when the vehicle state is the stop state, the provisional end-abutting counter **97** increments a count value by one.

[0208] The end-abutting counter **96**oL and the provisional end-abutting counter **97**oL are counters that count the number of times that the end-abutting steering angle range determination unit **91** determines that variation in left end-abutting steering angles θ abt is less than or equal to the predetermined threshold value θ tha when end-abutting occurs in the case of steering to the left side and no end-abutting occurs in the case of steering to the right side.

[0209] In addition, the end-abutting counter 96oR and the provisional end-abutting counter 97oR are counters that count the number of times that the end-abutting steering angle range determination unit 91 determines that variation in right end-abutting steering angles θ abt is less than or equal to the predetermined threshold value θ tha3 when no end-abutting occurs in the case of steering to the left side and end-abutting occurs in the case of steering to the right side. [0210] In addition, the end-abutting counter 96bL and the provisional end-abutting counter 97bL are counters that count the number of times that the end-abutting steering angle range determination unit 91 determines that variation in left end-abutting steering angles θ abt is less than or equal to the predetermined threshold value θ tha3 when end-abutting occurs both in the case of steering to the left side and in the case of steering to the right side.

[0211] In addition, the end-abutting counter **96**bR and the provisional end-abutting counter **97**bR are counters that count the number of times that the end-abutting steering angle range determination unit **91** determines that variation in right end-abutting steering angles θ abt is less than or equal to the predetermined threshold value θ tha**3** when end-abutting occurs both in the case of steering to the left side and in the case of steering to the right side.

[0212] When the vehicle state transitions from the stop state to the travel state, the counting unit **92** assigns a count value in the provisional end-abutting counter **97** to a count value in the end-abutting counter **96**. That is, the counting unit **92** replaces the count value in the end-abutting counter **96** by the count value in the provisional end-abutting counter **97**. In addition, the counting unit **92** resets the count value in the provisional end-abutting counter **97** to "0".

[0213] FIG. **14**C is now referred to. When a latest end-abutting steering angle θ abt (a triangular mark) is closer to a neutral position of the steering mechanism than stored end-abutting steering angles θ abt (circular marks) and a difference between a maximum value θ max (that is, an end-abutting steering angle θ abt farthest from the neutral position) of the stored end-abutting steering angles θ abt (the circular marks) and the latest end-abutting steering angle θ abt (the triangular mark) is greater than the predetermined threshold value θ tha**3**, the end-abutting steering angle range determination unit **91** determines that variation in the end-abutting steering angles θ abt is not less than or equal to the predetermined threshold value θ tha**3**. Hereinafter, such a state is sometime referred to as "neutral-side deviation state".

[0214] When the latest end-abutting steering angle θ abt (the triangular mark) is in the neutral-side deviation state, the end-abutting steering angle range determination unit **91** discards the latest end-abutting steering angle θ abt (the triangular mark).

[0215] In addition, the on-center counter $\bf 98$ counts the number of times that a latest end-abutting steering angle θ abt (a triangular mark) is determined to be in the neutral-side deviation state and store the counted number. That is, when the end-abutting steering angle range determination unit $\bf 91$ determines that a latest end-abutting steering angle θ abt (a triangular mark) is in the neutral-side deviation state, the on-center counter $\bf 98$ increments a count value by one.

[0216] The on-center counter **98**oL is a counter that counts the number of times that the end-abutting steering angle range determination unit **91** determines that a left end-abutting steering angle θ abt is in the neutral-side deviation state when end-abutting occurs in the case of steering to the left side and no end-abutting occurs in the case of steering to the right side.

[0217] The on-center counter **98**oR is a counter that counts the number of times that the end-abutting steering angle range determination unit **91** determines that a right end-abutting steering angle θ abt is in the neutral-side deviation state when no end-abutting occurs in the case of steering to the left side and end-abutting occurs in the case of steering to the right side.

[0218] The on-center counter **98**bL is a counter that counts the number of times that the end-abutting steering angle range determination unit **91** determines that a left end-abutting steering angle θ abt is in the neutral-side deviation state when end-abutting occurs both in the case of steering to the left side and in the case of steering to the right side.

[0219] The on-center counter **98**bR is a counter that counts the number of times that the endabutting steering angle range determination unit **91** determines that a right end-abutting steering angle θ abt is in the neutral-side deviation state when end-abutting occurs both in the case of steering to the left side and in the case of steering to the right side.

[0220] When a count value in one of the on-center counters **98**oL, **98**oR, **98**bL, and **98**bR becomes greater than or equal to a predetermined threshold value, the end-abutting steering angle range determination unit **91** deletes left or right end-abutting steering angles θabt (circular marks) stored in the end-abutting steering angle storage unit **94**. In addition, the counting unit **92** resets the count values in the end-abutting counters **96**oL, **96**oR, **96**bL, and **96**bR, the provisional end-abutting counters **97**oL, **97**oR, **97**bL, and **97**bR, and the on-center counters **98**oL, **98**oR, **98**bL, and **98**bR to "0".

[0221] FIG. **14**B is now referred to. When an end-abutting steering angle θ abt (a triangular mark) acquired most recently is located farther from the neutral position of the steering mechanism than stored end-abutting steering angles θ abt (circular marks) and a difference between a minimum value θ min (that is, an end-abutting steering angle θ abt closest to the neutral position) of the stored end-abutting steering angles θ abt (the circular marks) and the end-abutting steering angle θ abt (the triangular mark) acquired most recently is greater than the predetermined threshold value θ tha θ 0 than the end-abutting steering angles θ 0 abt is not less than or equal to the predetermined threshold value θ 1 than a state is sometime referred to as "end-side deviation state".

[0222] When a latest end-abutting steering angle θ abt (a triangular mark) is in the end-side

deviation state, the end-abutting steering angle range determination unit 91 deletes end-abutting steering angles θ abt (circular marks) stored at time points before the latest end-abutting steering angle θ abt (the triangular mark) is acquired from the end-abutting steering angle storage unit 94 and stores the latest end-abutting steering angle θ abt (the triangular mark) in the end-abutting steering angle storage unit 94.

[0223] In addition, the counting unit **92** resets count values in the end-abutting counter **96**, the provisional end-abutting counter **97**, and the on-center counter **98** to "0". When the vehicle state is the travel state, the counting unit **92** increments the end-abutting counter **96** by one, and when the vehicle state is the stop state, the counting unit **92** increments the provisional end-abutting counter **97** by one.

[0224] Specifically, when end-abutting occurs in the case of steering to the left side and no end-abutting occurs in the case of steering to the right side and the latest end-abutting steering angle θ abt (the triangular mark) is a left end-abutting steering angle θ abt, the end-abutting steering angle range determination unit **91** deletes stored left end-abutting steering angles θ abt (circular marks) from the end-abutting steering angle storage unit **94** and stores the latest left end-abutting steering angle θ abt (the triangular mark) in the end-abutting steering angle storage unit **94**. In addition, the counting unit **92** resets the count values in the end-abutting counter **96**oL, the provisional end-abutting counter **97**oL, and the on-center counter **98**oL to "0". When the vehicle state is the travel state, the counting unit **92** increments the end-abutting counter **96**oL by one, and when the vehicle state is the stop state, the counting unit **92** increments the provisional end-abutting counter **97**oL by one.

[0225] In addition, when no end-abutting occurs in the case of steering to the left side and end-abutting occurs in the case of steering to the right side and the latest end-abutting steering angle θ abt (the triangular mark) is a right end-abutting steering angle θ abt, the end-abutting steering angle range determination unit θ 1 deletes stored right end-abutting steering angles θ abt (circular marks) from the end-abutting steering angle storage unit θ 4 and stores the latest right end-abutting steering angle θ abt (the triangular mark) in the end-abutting steering angle storage unit θ 4. In addition, the counting unit θ 2 resets the count values in the end-abutting counter θ 6oR, the provisional end-abutting counter θ 7oR, and the on-center counter θ 8oR to "0". When the vehicle state is the travel state, the counting unit θ 2 increments the end-abutting counter θ 6oR by one, and when the vehicle state is the stop state, the counting unit θ 2 increments the provisional end-abutting counter θ 7oR by one.

[0226] In addition, when end-abutting occurs both in the case of steering to the left side and in the case of steering to the right side and the latest end-abutting steering angle θ abt (the triangular mark) is a left end-abutting steering angle θ abt, the end-abutting steering angle range determination unit **91** deletes stored left end-abutting steering angles θ abt (circular marks) from the end-abutting steering angle storage unit 94 and stores the latest left end-abutting steering angle θ abt (the triangular mark) in the end-abutting steering angle storage unit 94. In addition, the counting unit 92 resets the count values in the end-abutting counter **96**bL, the provisional end-abutting counter **97**bL, and the on-center counter **98**bL to "0". When the vehicle state is the travel state, the counting unit **92** increments the end-abutting counter **96**bL by one, and when the vehicle state is the stop state, the counting unit **92** increments the provisional end-abutting counter **97**bL by one. [0227] In addition, when end-abutting occurs both in the case of steering to the left side and in the case of steering to the right side and the latest end-abutting steering angle θ abt (the triangular mark) is a right end-abutting steering angle θ abt, the end-abutting steering angle range determination unit **91** deletes stored right end-abutting steering angles θ abt (circular marks) from the end-abutting steering angle storage unit **94** and stores the latest right end-abutting steering angle θ abt (the triangular mark) in the end-abutting steering angle storage unit **94**. In addition, the counting unit **92** resets the count values in the end-abutting counter **96**bR, the provisional end-abutting counter 97bR, and the on-center counter 98bR to "0". When the vehicle state is the travel state, the

counting unit **92** increments the end-abutting counter **96**bR by one, and when the vehicle state is the stop state, the counting unit **92** increments the provisional end-abutting counter **97**bR by one. [0228] When end-abutting occurs both in the case of steering to the left side and in the case of steering to the right side, the counting unit **92** receives a determination result of comparison between a rack stroke approximate value Sta (FIG. **13**B) that the end-abutting detection unit **90** calculates, as an absolute value of a difference between the left and right end-abutting steering angles θabt and the erroneous stroke minimum value Sterr, from the end-abutting detection unit **90**. [0229] When the rack stroke approximate value Sta is less than the erroneous stroke minimum value Sterr, the counting unit **92** determines that false detection of an end-abutting steering angle θabt has occurred and saves the count values in the end-abutting counters **96**bL and **96**bR in the provisional end-abutting counters **97**bL and **97**bR. That is, the counting unit **92** replaces the count values in the provisional end-abutting counters **97**bL and **97**bR by the count values in the end-abutting counters **96**bL and **96**bR, respectively and resets the count values in the end-abutting counters **96**bL and **96**bR to "0".

[0230] Subsequently, when the vehicle state transitions to the travel state in the determination processing of the vehicle state and the detection of the end-abutting steering angle θabt is determined to be reliable, the count values in the provisional end-abutting counters **97**bL and **97**bR are returned to the end-abutting counters **96**bL and **96**bR, respectively.

[0231] Note that variation in a latest end-abutting steering angle θ abt (a triangular mark) and stored end-abutting steering angles θ abt (circular marks) being less than or equal to the predetermined threshold value θ tha3 as illustrated in FIG. 14A is sometimes referred to as "an end-abutting steering angle θ abt acquired most recently falls within the predetermined variation range θ tha3" in the following description. In addition, a latest end-abutting steering angle θ abt (a triangular mark) being in the end-side deviation state or the neutral-side deviation state as illustrated in FIG. 14B or FIG. 14C is sometimes referred to as "an end-abutting steering angle θ abt acquired most recently falls outside the predetermined variation range θ tha3" in the following description.

[0232] FIG. **11** is now referred to. The relearning unit **93** determines necessity of relearning of one of the virtual rack end positions θ ev, based on a count value in the end-abutting counter **96**, and when the relearning of the virtual rack end position θ ev is to be performed, the relearning unit **93** outputs a relearning command signal Cmd to the terminal position learning unit **46**.

[0233] For example, when a count value in the end-abutting counter **96** exceeds a predetermined threshold value, the relearning unit **93** determines that relearning of a corresponding virtual rack end position θ ev is necessary and outputs a relearning command signal Cmd.

[0234] For example, when end-abutting occurs in the case of steering to the left side and no end-abutting occurs in the case of steering to the right side and the count value in the end-abutting counter **96**oL for the left side exceeds the predetermined threshold value, the relearning unit **93** may output a relearning command signal Cmd commanding relearning of the left virtual rack end position θ evl to the terminal position learning unit **46**. The terminal position learning unit **46** resets the left virtual rack end position θ evl to the initial value $-\theta$ int in accordance with the relearning command signal Cmd. Because of this configuration, the terminal position learning unit **46** relearns the left virtual rack end position θ evl.

[0235] In addition, for example, when no end-abutting occurs in the case of steering to the left side and end-abutting occurs in the case of steering to the right side and the count value in the end-abutting counter **96**oR for the right side exceeds the predetermined threshold value, the relearning unit **93** may output a relearning command signal Cmd commanding relearning of the right virtual rack end position θ evr to the terminal position learning unit **46**. The terminal position learning unit **46** resets the right virtual rack end position θ evr to the initial value θ int in accordance with the relearning command signal Cmd. Because of this configuration, the terminal position learning unit **46** relearns the right virtual rack end position θ evr.

[0236] In addition, for example, when end-abutting occurs both in the case of steering to the left

side and in the case of steering to the right side and a sum of the count value in the end-abutting counter **96**bL for the left side and the count value in the end-abutting counter **96**bR for the right side is greater than or equal to a predetermined threshold value, the relearning unit **93** calculates the rack stroke minimum value Stmin, the rack stroke maximum value Stmax, the learning threshold value θ lth, the rack end maximum value θ evmax, and the initial value θ int, which are set values used for the learning of the virtual rack end positions θ ev, based on end-abutting steering angles θ abt stored in the end-abutting steering angle storage unit **94** of the end-abutting steering angle range determination unit **91**.

[0237] With reference to FIG. **16**, an example of a method for calculating the rack stroke minimum value Stmin, the rack stroke maximum value Stmax, the learning threshold value θ lth, the rack end maximum value θ evmax, and the initial value θ int will be described. Note that in FIG. **16**, a reference sign "RC" denotes a center position between the left and right rack ends, and "0 [deg]" denotes a position at which the steering angle θ h detected by the steering angle sensor **14** is "0". [0238] The relearning unit **93** calculates a sum of an absolute value of a maximum value of left end-abutting steering angles θ abt (that is, a left end-abutting steering angle θ abt among the left end-abutting steering angles θ abt that is closest to the neutral position) and an absolute value of a minimum value of right end-abutting steering angles θ abt (that is, a right end-abutting steering angle θ abt among the right end-abutting steering angles θ abt that is closest to the neutral position) as a provisional stroke Stp=|max (left end-abutting steering angles θ abt)|+|min (right end-abutting steering angles θ abt)|.

[0239] Next, the relearning unit **93** calculates a half of an absolute value of a difference between the absolute value of the maximum value of the left end-abutting steering angles θ abt and the absolute value of the minimum value of the right end-abutting steering angles θ abt as a provisional assembly error $e\theta = \|\max (\text{left end-abutting steering angles }\theta$ abt) $\|-\|$ min (right end-abutting steering angles θ abt) $\|-\|$ 2.

[0240] The relearning unit **93** calculates a subtraction result obtained by subtracting predetermined thermal expansion/contraction error of a manual steering gear and learning error from the provisional stroke Stp as a rack stroke minimum value Stmin=Stp-(thermal expansion/contraction error of manual steering gear)-(learning error).

[0241] The relearning unit **93** calculates an addition result obtained by adding the predetermined thermal expansion/contraction error of the manual steering gear, the learning error, and sensor error to the provisional stroke Stp as a rack stroke maximum value Stmax=Stp+(thermal expansion/contraction error of manual steering gear)+(learning error)+(sensor error).

[0242] The relearning unit **93** calculates a sum of a half of the rack stroke maximum value Stmax and the provisional assembly error $e\theta$ as a rack end maximum value $\theta = \frac{(Stmax/2 + e\theta)}{(Stmax/2 + e\theta)}$.

[0243] The relearning unit **93** calculates a subtraction result obtained by subtracting the rack end maximum value θ evmax from the rack stroke minimum value Stmin as an initial value θ int= (Stmin- θ evmax).

[0244] The relearning unit **93** calculates a subtraction result obtained by subtracting the provisional assembly error $e\theta$ from a half of the rack stroke minimum value Stmin as a learning threshold value θ lth=Stmin/2-(provisional assembly error $e\theta$).

[0245] The relearning unit **93** outputs the calculated initial value θ int and rack stroke maximum value Stmax to the terminal position learning unit **46**. In addition, the relearning unit **93** outputs the calculated rack stroke maximum value Stmax, rack stroke minimum value Stmin, and learning threshold value θ 1th to the learning state determination unit **51**. The relearning unit **93** may be configured to be able to output the rack end maximum value θ 2vmax.

[0246] The terminal position learning unit **46** updates the initial values θ int of the virtual rack end positions θ evr and θ evl at the time of learning start to values that the relearning unit **93** recalculated. The terminal position learning unit **46** updates the rack stroke maximum value Stmax to be used for determination of offset error to a value that the relearning unit **93** recalculated.

[0247] In addition, the learning state determination unit **51** updates the rack stroke maximum value Stmax, the rack stroke minimum value Stmin, and the learning threshold value θ th to be used for determination of a learning state to values that the relearning unit **93** recalculated.

[0248] The relearning unit **93** may output a relearning command signal Cmd commanding relearning of the left virtual rack end position θ evl and the right virtual rack end position θ evr to the terminal position learning unit **46**. The terminal position learning unit **46** resets the left virtual rack end position θ evl to the initial value θ int in accordance with the relearning command signal Cmd. Because of this configuration, the terminal position learning unit **46** relearns the left virtual rack end position θ evl and the right virtual rack end position θ evr. (Operation)

[0249] FIG. **17** is a flowchart of an example of a turning control method of the embodiment.

[0250] In step S1, the end-abutting detection unit $\bf 90$ retains a steering angle θ dt when the steering torque Th is greater than or equal to a predetermined threshold value Tth1.

[0251] In step S2, the end-abutting detection unit **90** determines whether or not an absolute value $|\theta ev - \theta dt|$ of a difference between the retained steering angle θdt and one of the virtual rack end positions θev is greater than the steering angle threshold value θdt . When the absolute value $|\theta ev - \theta dt|$ is not greater than the steering angle threshold value θdt (step S2: N), the process returns to step S1. When the absolute value $|\theta ev - \theta dt|$ is greater than the steering angle threshold value θdt (step S2: Y), the process proceeds to step S3.

[0252] In step S3, the end-abutting detection unit $\bf 90$ retains an end-abutting steering angle candidate θcd when, after the steering torque Th has become greater than or equal to the predetermined threshold value Tth1, the column output shaft torque Tc becomes less than the predetermined threshold value Tth2 and the steering torque Th becomes less than the predetermined threshold value Tth1.

[0253] In step S4, the end-abutting detection unit **90** determines whether or not an absolute value $|\theta ev-\theta cd|$ of a difference between an end-abutting steering angle candidate θcd and the virtual rack end position θev is greater than the steering angle threshold value $\theta tha2$. When the absolute value $|\theta ev-\theta cd|$ is not greater than the steering angle threshold value $\theta tha2$ (step S4: N), the process returns to step S1. When the absolute value $|\theta ev-\theta cd|$ is greater than the steering angle threshold value $\theta tha2$ (step S4: Y), the process proceeds to step S5.

[0254] In step S5, the vehicle state determination unit 95 determines whether the vehicle is in the stop state or the travel state. When the vehicle state transitions from the stop state to the travel state, the counting unit 92 assigns a count value in the provisional end-abutting counter 97 to count value in the end-abutting counter 96. In addition, the counting unit 92 resets the count value in the provisional end-abutting counter 97 to "0".

[0255] In step S6, the end-abutting detection unit 90 determines whether or not end-abutting occurs both in the case of the steering wheel 1 being steered to the left side and in the case of being steered to the right side. When end-abutting occurs both in the case of steering to the left side and in the case of steering to the right side (step S6: Y), the process proceeds to step S8. When end-abutting occurs only either in the case of steering to the left side or in the case of steering to the right side (step S6: N), the process proceeds to step S7.

[0256] In step S7, the relearning determination unit **53** performs one-side end-abutting processing. Details of the one-side end-abutting processing will be described later with reference to FIG. **18**. Subsequently, the process proceeds to step S**9**.

[0257] In step S**8**, the relearning determination unit **53** performs both-side end-abutting processing. Details of the both-side end-abutting processing will be described later with reference to FIG. **19**. Subsequently, the process proceeds to step S**9**.

[0258] In step **S9**, the controller **30** determines whether or not the IGN key **11** is turned off. When the IGN key **11** is not turned off (step **S9**: N), the process returns to step **S1**. When the IGN key **11**

is turned off (step S9: Y), the process terminates.

[0259] FIG. **18** is a flowchart of an example of the one-side end-abutting processing (S7) in FIG. **17**. The one-side end-abutting processing is performed independently when end-abutting occurs in the case of steering to the right side (that is, a right end-abutting steering angle θ abt is acquired) and when end-abutting occurs in the case of steering to the left side (that is, a left end-abutting steering angle θ abt is acquired).

[0260] In step S10, the end-abutting detection unit 90 calculates a rack stroke approximate value Sta, based on an end-abutting steering angle θ abt acquired most recently. For example, when a right end-abutting steering angle θ abt is acquired, the end-abutting detection unit 90 calculates an absolute value of a difference between the left virtual rack end position θ evl and the right end-abutting steering angle θ abt as the rack stroke approximate value Sta. When a left end-abutting steering angle θ abt is acquired, the end-abutting detection unit 90 calculates an absolute value of a difference between the right virtual rack end position θ evr and the left end-abutting steering angle θ abt as the rack stroke approximate value Sta. When error between the rack stroke approximate value Sta and a predetermined value does not fall within a threshold value (step S10: N), the one-side end-abutting processing is terminated. When the error between the rack stroke approximate value Sta and the predetermined value falls within the threshold value (step S10: Y), the process proceeds to step S11.

[0261] In step S11, the end-abutting steering angle range determination unit 91 determines whether or not an end-abutting steering angle θ abt acquired most recently falls within a predetermined variation range θ tha3. When the end-abutting steering angle θ abt acquired most recently falls outside the predetermined variation range θ tha3 (step S11: N), the process proceeds to step S15. When the end-abutting steering angle θ abt acquired most recently falls within the predetermined variation range θ tha3 (step S11: Y), the process proceeds to step S12.

[0262] In step S12, the counting unit 92 increments a count value in the end-abutting counter 96 or the provisional end-abutting counter 97 by one. Specifically, when the vehicle is in the travel state and a left end-abutting steering angle θ abt is acquired, the counting unit 92 increments a count value in the end-abutting steering angle θ abt is acquired, the counting unit 92 increments a count value in the end-abutting counter 96 oR by one. When the vehicle is in the stop state and a left end-abutting steering angle θ abt is acquired, the counting unit 92 increments a count value in the end-abutting counter 97 oL by one. When the vehicle is in the stop state and a right end-abutting steering angle θ abt is acquired, the counting unit 92 increments a count value in the end-abutting steering angle θ abt is acquired, the counting unit 92 increments a count value in the end-abutting counter 97 oR by one.

[0263] In step S13, the relearning unit 93 determines whether or not a count value in the endabutting counters 96 exceeds a predetermined threshold value. When a count value in the endabutting counter 96 does not exceed the predetermined threshold value (step S13: N), the one-side end-abutting processing is terminated. When a count value in the end-abutting counter 96 exceeds the predetermined threshold value (step S13: Y), the process proceeds to step S14.

[0264] In step S14, the relearning unit 93 outputs a relearning command signal Cmd to the terminal position learning unit 46.

[0265] Specifically, when the count value in the left end-abutting counter **96**oL exceeds the predetermined threshold value, the relearning unit **93** outputs a relearning command signal Cmd commanding relearning of the left virtual rack end position θ evl. The terminal position learning unit **46** resets the left virtual rack end position θ evl to an initial value-fint in accordance with the relearning command signal Cmd. When the count value in the right end-abutting counter **96**oR exceeds the predetermined threshold value, the relearning unit **93** outputs a relearning command signal Cmd commanding relearning of the right virtual rack end position θ evr. The terminal position learning unit **46** resets the right virtual rack end position θ evr to an initial value θ int in accordance with the relearning command signal Cmd. Subsequently, the one-side end-abutting

processing is terminated.

[0266] In step S15, the end-abutting steering angle range determination unit 91 determines whether or not the end-abutting steering angle θ abt acquired most recently is in the neutral-side deviation state. When the end-abutting steering angle θ abt acquired most recently is in the end-side deviation state (step S15: N), the process proceeds to step S19. When the end-abutting steering angle θ abt acquired most recently is in the neutral-side deviation state (step S15: Y), the process proceeds to step S16.

[0267] In step S16, the counting unit 92 increments a count value in the on-center counter 98 by one. Specifically, when the end-abutting steering angle θ abt acquired most recently is a left endabutting steering angle θ abt, the counting unit **92** increments the count value in the on-center counter **98**oL by one, and when the end-abutting steering angle θabt acquired most recently is a right end-abutting steering angle θ abt, the counting unit **92** increments the count value in the oncenter counter **98**oR by one.

[0268] In step S17, the counting unit 92 determines whether or not the count value in either the oncenter counters **98**oL or **98**oR is greater than or equal to the predetermined threshold value. When neither count value is greater than or equal to the threshold value (step S17: N), the one-side endabutting processing is terminated. When the count value in either the end-abutting counters **98**oL or **98**oR is greater than or equal to the threshold value (step S17: Y), the process proceeds to step S18. [0269] In step S18, the counting unit 92 resets the count values in the end-abutting counters 96oL, 96oR, 96bL, and 96bR, the provisional end-abutting counters 97oL, 97oR, 97bL, and 97bR, and the on-center counters 98oL, 98oR, 98bL, and 98bR to "0". In addition, the end-abutting steering angle range determination unit 91 deletes the left or right end-abutting steering angles θ abt stored in the end-abutting steering angle storage unit 94. Subsequently, the one-side end-abutting processing is terminated.

[0270] In step S19, the counting unit 92 resets the end-abutting counter 96oL and the provisional end-abutting counter **97**oL or the end-abutting counter **96**oR and the provisional end-abutting counter **97**oR to "0", depending on which one of left and right end-abutting steering angles θ abt is in the end-side deviation state. In addition, the end-abutting steering angle range determination unit **91** deletes one of the left and right end-abutting steering angles θ abt stored in the end-abutting steering angle storage unit **94**. Specifically, when a left end-abutting steering angle θ abt is in the end-side deviation state, the counting unit 92 resets the end-abutting counter 96oL and the provisional end-abutting counter **97**oL to "0" and the end-abutting steering angle range determination unit **91** deletes the left end-abutting steering angle θ abt. When a right end-abutting steering angle θ abt is in the end-side deviation state, the counting unit **92** resets the end-abutting counter **96**oR and the provisional end-abutting counter **97**oR to "0" and the end-abutting steering angle range determination unit **91** deletes the right end-abutting steering angle θ abt. [0271] Processing in step S20 is the same as the processing in step S12. Subsequently, the one-side

end-abutting processing is terminated.

[0272] FIG. **19** is a flowchart of an example of the both-side end-abutting processing (**S8**) in FIG. **17**.

[0273] In step S**30**, the end-abutting steering angle range determination unit **91** determines whether or not an end-abutting steering angle θ abt acquired most recently falls within a predetermined variation range θ tha3. When the end-abutting steering angle θ abt acquired most recently falls outside the predetermined variation range θ tha3 (step S30: N), the process proceeds to step S36. Note that processing in steps S**36** to S**42** is performed independently when end-abutting occurs in the case of steering to the right side (that is, a right end-abutting steering angle θ abt is acquired) and when end-abutting occurs in the case of steering to the left side (that is, a left end-abutting steering angle θ abt is acquired).

[0274] In contrast, when the end-abutting steering angle θ abt acquired most recently falls within the predetermined variation range θ tha3 (step S30: Y), the process proceeds to step S31.

[0275] In step S31, the counting unit 92 increments a count value in the end-abutting counter 96 or the provisional end-abutting counter 97 by one. Specifically, when the vehicle is in the travel state and a left end-abutting steering angle θ abt is acquired, the counting unit 92 increments a count value in the end-abutting steering angle θ abt is acquired, the counting unit 92 increments a count value in the end-abutting counter 96bR by one. Specifically, when the vehicle is in the stop state and a left end-abutting steering angle θ abt is acquired, the counting unit 92 increments a count value in the end-abutting counter 97bL by one. When the vehicle is in the stop state and a right end-abutting steering angle θ abt is acquired, the counting unit 92 increments a count value in the end-abutting counter 97bR by one.

[0276] In step S32, the end-abutting detection unit 90 calculates an absolute value of a difference between the left and right end-abutting steering angles θ abt as a rack stroke approximate value Sta. For example, the end-abutting detection unit 90 may calculate an absolute value of a difference between a minimum value of stored right end-abutting steering angles θ abt (that is, a right end-abutting steering angle θ abt among the stored right end-abutting steering angles θ abt that is closest to the neutral position) and a maximum value of stored left end-abutting steering angles θ abt (that is, a left end-abutting steering angle θ abt among the stored left end-abutting steering angles θ abt that is closest to the neutral position) as a rack stroke approximate value Sta=|min (right end-abutting steering angles θ abt)|-max (left end-abutting steering angles θ abt)|. When the rack stroke approximate value Sta is not greater than or equal to the erroneous stroke minimum value Sterr (step S32: N), the process proceeds to step S35. When the rack stroke approximate value Sta is greater than or equal to the erroneous stroke minimum value Sterr (step S32: Y), the process proceeds to step S33.

[0277] In step S33, the relearning unit 93 determines whether or not a sum of the count value in the end-abutting counter 96bL and the count value in the end-abutting counter 96bR exceeds a predetermined threshold value. When the sum of the count values is not greater than or equal to the threshold value (step S33: N), the both-side end-abutting processing is terminated. When the sum of the count values is greater than or equal to the threshold value (step S33: Y), the process proceeds to step S34.

[0278] In step S34, the relearning unit 93 calculates setting values to be used for learning of the virtual rack end positions θ ev (the rack stroke minimum value Stmin, the rack stroke maximum value Stmax, the learning threshold value θ lth, the rack end maximum value θ evmax, and the initial value θ int). The relearning unit 93 outputs the calculated initial value θ int and rack stroke maximum value Stmax to the terminal position learning unit 46 and outputs the calculated rack stroke maximum value Stmax, rack stroke minimum value Stmin, and learning threshold value θ lth to the learning state determination unit 51. The terminal position learning unit 46 and the learning state determination unit 51 update the setting values to values received from the relearning unit 93. [0279] The relearning unit 93 outputs a relearning command signal Cmd to the terminal position learning unit 46 resets the left virtual rack end position θ evl to the initial value θ int and resets the right virtual rack end position θ evr to the initial value θ int in accordance with the relearning command signal Cmd. Subsequently, the both-side end-abutting processing is terminated.

[0280] In step S35, the counting unit 92 saves the count values in the end-abutting counters 96bL and 96bR in the provisional end-abutting counters 97bL and 97bR. Subsequently, the both-side end-abutting processing is terminated.

[0281] In step S**36**, the end-abutting detection unit **90** calculates an absolute value of a difference between the left and right end-abutting steering angles θabt as a rack stroke approximate value Sta. When the rack stroke approximate value Sta is not greater than or equal to the erroneous stroke minimum value Sterr (step S**36**: N), the both-side end-abutting processing is terminated. When the rack stroke approximate value Sta is greater than or equal to the erroneous stroke minimum value

Sterr (step S**36**: Y), the process proceeds to step S**37**.

[0282] In step S37, the end-abutting steering angle range determination unit 91 determines whether or not the end-abutting steering angle θ abt acquired most recently is in the neutral-side deviation state. When the end-abutting steering angle θ abt acquired most recently is in the end-side deviation state (step S37: N), the process proceeds to step S41. When the end-abutting steering angle θ abt acquired most recently is in the neutral-side deviation state (step S37: Y), the process proceeds to step S38.

[0283] In step S38, the counting unit 92 increments a count value in the on-center counter 98 by one. Specifically, when the end-abutting steering angle θ abt acquired most recently is a left end-abutting steering angle θ abt, the counting unit 92 increments the count value in the on-center counter 98bL by one, and when the end-abutting steering angle θ abt acquired most recently is a right end-abutting steering angle θ abt, the counting unit 92 increments the count value in the on-center counter 98bR by one.

[0284] In step S39, the counting unit 92 determines whether or not the count value in either the oncenter counters 98bL or 98bR is greater than or equal to the predetermined threshold value. When neither count value is greater than or equal to the threshold value (step S39: N), the both-side end-abutting processing is terminated. When the count value in either the end-abutting counters 98oL or 98oR is greater than or equal to the threshold value (step S38: Y), the process proceeds to step S40. [0285] Processing in step S40 is the same as the processing in step S18 in FIG. 18. Subsequently, the both-side end-abutting processing is terminated.

[0286] In step S41, the counting unit 92 resets either the end-abutting counter 96bL and the provisional end-abutting counter 97bL or the end-abutting counter 96bR and the provisional end-abutting counter 97bR to "0", depending on which one of the left and right end-abutting steering angle θ and the end-side deviation state. In addition, the end-abutting steering angle range determination unit 91 deletes one of the left and right end-abutting steering angles θ abt stored in the end-abutting steering angle storage unit 94. Specifically, when the left end-abutting steering angle θ and the provisional end-abutting counter 97bL to "0" and the end-abutting steering angle range determination unit 91 deletes the left end-abutting steering angle θ abt. When the right end-abutting steering angle θ abt is in the end-side deviation state, the counting unit 92 resets the end-abutting counter 96bR and the provisional end-abutting counter 97bR to "0" and the end-abutting steering angle range determination unit 91 deletes the right end-abutting steering angle θ abt. [0287] Processing in step S42 is the same as the processing in step S31. Subsequently, the both-side end-abutting processing is terminated.

(Variations)

[0288] Although an embodiment in which the turning control device of the present invention is applied to the electric power steering device is described above, the turning control device of the present invention is widely applicable to a variety of turning control devices other than the electric power steering device as long as the turning control device is a turning control device that generates force to turn the steered wheels of a vehicle by an actuator. For example, the turning control device of the present invention may be applied to a steering device of a steer-by-wire (SBW) type in which the steering wheel and the steered wheels are mechanically separated from each other. In this case, the steering torque Th does not have to be added to the motor torque when the column output shaft torque Tc is calculated.

[0289] (1) The end-abutting detection unit **90** may correct an end-abutting steering angle θ abt according to the amount of deformation of a mechanical part due to the column output shaft torque Tc. This configuration enables influence of torsion of the turning mechanism to be reduced. The amount of deformation can be calculated from the column output shaft torque Tc and rigidity (spring constant) of the mechanical part. The rigidity is a ratio of a change amount of the column output shaft torque Tc to a change amount of the steering angle θ h at a point indicated by the

reference sign $\theta 1$ or $\theta 2$ in FIGS. 7A and 7B and may be acquired through an experiment or the like. [0290] Characteristics illustrated in FIGS. 7A and 7B changes due to viscous resistance, which depends on steering velocity. There is a possibility that the steering velocity influences a learned value of a rack end. It may be configured such that a change amount of the learned value of the rack end with respect to a change amount of the steering velocity is acquired through an experiment or the like and stored as steering velocity-correction amount characteristics and the end-abutting steering angle θ abt is corrected based on the steering velocity and the steering velocity-correction amount characteristics.

[0291] (2) In the above-described embodiment, when the rack stroke St between the right and left virtual rack end positions θ ev exceeds the rack stroke maximum value Stmax, a virtual rack end position θ ev of a stroke end where no end-abutting is detected is reset in such a way that the rack stroke St coincides with the predetermined rack stroke minimum value. In place of the configuration, when one of the right and left virtual rack end positions θ ev exceeds the rack end maximum value θ evmax, the other of the right and left virtual rack end positions θ ev may be reset in such a way that the rack stroke St coincides with the predetermined rack stroke minimum value. Advantageous Effects of Embodiment

[0292] (1) A turning control device includes: a position detection unit configured to detect a steered position of a turning mechanism of a vehicle; a terminal position learning unit configured to learn a terminal position of the turning mechanism, based on the steered position detected by the position detection unit; and a relearning determination unit configured to determine necessity of relearning of the terminal position. The relearning determination unit includes: an end-abutting detection unit configured to detect occurrence of end-abutting, the end-abutting being a state in which the turning mechanism is turned to the terminal position, and acquire an end-abutting steering angle, the end-abutting steering angle being a steering angle when occurrence of end-abutting is detected; an end-abutting steering angle range determination unit configured to determine whether or not variation in the end-abutting steering angles acquired multiple times is less than or equal to a predetermined threshold value; and a relearning unit configured to, when the variation is less than or equal to the predetermined threshold value, reset the learned terminal position to an initial value.

[0293] Because of this configuration, it is possible to prevent mistakenly detecting end-abutting due to rapid turning-back steering, collision of a tire with a curb, or the like before end-abutting actually occurs and relearning a terminal position. Thus, incorrect learning of a terminal position can be prevented.

[0294] (2) When occurrence of end-abutting is detected both in a case of steering to a left side and in a case of steering to a right side and each of variation in the end-abutting steering angles in the case of steering to the left side and variation in the end-abutting steering angles in the case of steering to the right side is less than or equal to the predetermined threshold value, the relearning unit may calculate estimated rack stroke, based on the end-abutting steering angles and reset the initial value, based on the estimated rack stroke.

[0295] Because of this configuration, when a rack shaft having a length different from length of a rack shaft that should be originally mounted is mistakenly mounted, a learning initial value can be set according to rack length of the rack shaft that is actually mounted.

[0296] (3) The turning control device may include a counting unit configured to count a number of times of acquisition of the end-abutting steering angles having variation less than or equal to the predetermined threshold value, and when the number of times of acquisition is greater than or equal to a first threshold number of times, the relearning unit may reset the learned terminal position to the initial value.

[0297] Because of this configuration, whether or not variation in the end-abutting steering angles is less than or equal to a predetermined threshold value can be determined more accurately.

[0298] (4) The steering angle range determination unit may store the acquired end-abutting steering

angle and determine whether or not variation in the end-abutting steering angles is less than or

equal to the predetermined threshold value, based on a difference between the stored end-abutting steering angle and a steering angle when occurrence of end-abutting is detected. The counting unit may reset counting of the number of times of acquisition when variation in the end-abutting steering angles is determined not to be less than or equal to the predetermined threshold value. [0299] Because of this configuration, mistakenly relearning a terminal position when variation in the end-abutting steering angles is temporarily less than or equal to a predetermined threshold value can be prevented.

[0300] (5) The counting unit may count a number of times that it is determined that a second steering angle, the second steering angle being a steering angle when occurrence of end-abutting is detected, is closer to a neutral position of the turning mechanism than a first steering angle, the first steering angle being a steering angle located farthest from the neutral position among the stored end-abutting steering angles, and a difference between the first steering angle and the second steering angle is not less than or equal to the predetermined threshold value, as a number of times of being on-center, and when the number of times of being on-center is greater than or equal to a second threshold number of times, reset counting of the number of times of acquisition.

[0301] Because of this configuration, mistakenly relearning a terminal position while the steering wheel cannot be steered to an angle at which end-abutting actually occurs because a tire is in contact with a curb or the like can be prevented.

[0302] (6) The counting unit may include a count storage unit configured to retain a number of times that the end-abutting steering angle having variation less than or equal to the predetermined threshold value is acquired during a period when vehicle speed of a vehicle is greater than or equal to a vehicle speed threshold value, as a counted number of times of end-abutting and retain a number of times that the end-abutting steering angle having variation less than or equal to the predetermined threshold value is acquired during a period when the vehicle speed is less than a vehicle speed threshold value, as a provisional counted number of times and, when the vehicle speed changes from less than the vehicle speed threshold value to greater than or equal to the vehicle speed threshold value, assign a value of the provisional counted number of times to the counted number of times of end-abutting. When the counted number of times of end-abutting is greater than or equal to the first threshold number of times, the relearning unit may reset the learned terminal position to an initial value.

[0303] There are some cases where in the stop state, there is a possibility that a tire is in contact with a curb or the like and an end-abutting steering angle is mistakenly acquired before end-abutting actually occurs. Because of this configuration, deferring relearning until the vehicle transitions from the stop state to the travel state enables incorrect learning to be prevented.

[0304] (7) The end-abutting detection unit may, when no end-abutting occurs in a case of steering to one of the left side and the right side and end-abutting occurs in a case of steering to the other of the left side and the right side, calculate a rack stroke, based on the learned terminal position learned on the one of the left side and the right side and the end-abutting steering angle acquired on the other of the left side and the right side. When error of the calculated rack stroke does not fall within a predetermined allowable range, the end-abutting detection unit does not use the end-abutting steering angle acquired on the other of the left side and the right side for calculation of variation in the end-abutting steering angles.

[0305] When error of the rack stroke calculated based on the end-abutting steering angles does not fall within a predetermined allowable range, there is a possibility of erroneous detection of an end-abutting steering angle. Not using such an end-abutting steering angle enables incorrect learning to be prevented.

[0306] (8) The end-abutting detection unit may, when end-abutting occurs both in a case of steering to the left side and in a case of steering to the right side, calculate rack stroke, based on the end-abutting steering angles acquired in the case of steering to the left side and in the case of steering to the right side, and when the calculated rack stroke is less than a predetermined lower limit, assign a

value of the counted number of times of end-abutting to the provisional counted number of times and resets a value of the counted number of times of end-abutting to 0.

[0307] When end-abutting occurs both in a case of steering to the left side and in a case of steering to the right side and the rack stroke calculated based on the end-abutting steering angles is less than a predetermined lower limit as described above, there is a possibility of erroneous detection of an end-abutting steering angle. Thus, moving a value of the counted number of times of end-abutting to the provisional counted number of times and temporarily deferring relearning enable incorrect learning to be prevented.

REFERENCE SIGNS LIST

[0308] **1** Steering wheel [0309] **2***i* Column input shaft [0310] **2***o* Column output shaft [0311] **3** Reduction gear [0312] **4** Intermediate shaft [0313] **4***a*, **4***b* Universal joint [0314] **4***c* Shaft member [0315] **5** Pinion rack mechanism [0316] **5***a* Pinion [0317] **5***b* Rack [0318] **6***a*, **6***b* Tie rod [0319] **7***a*, *7b* Hub unit [0320] **8**L, **8**R Steered wheel [0321] **10** Torque sensor [0322] **11** Ignition key [0323] **12** Vehicle speed sensor [0324] **13** Battery [0325] **14** Steering angle sensor [0326] **20** Motor [0327] **30** Controller [0328] **40** Basic command value calculation unit [0329] **41**, **64** Adder [0330] **42**, **82** Subtracter [0331] **43** Current control unit [0332] **44** PWM control unit [0333] **45** Inverter [0334] **46** Terminal position learning unit [0335] **47** Control rotational displacement setting unit [0336] **48** Differentiating unit [0337] **49** Impact mitigation control unit [0338] **50** Current detector [0339] **51** Learning state determination unit [0340] **52** Impact mitigation control output limiting unit [0341] **53** Relearning determination unit [0342] **60** Spring constant table [0343] **61**, **63** Multiplier [0344] **62** Viscosity constant table [0345] **65** Inverter [0346] **66**, **79** Limiter [0347] **70** Output shaft torque calculation unit [0348] 71 Selection unit [0349] 72 First storage unit [0350] 73, 77 Delay unit [0351] **74** Rate limiter [0352] **75** Corrected position calculation unit [0353] **76** Second storage unit [0354] **78** Third storage unit [0355] **80** Stroke calculation unit [0356] **81** Offset error calculation unit [0357] **83** Terminal position correction unit [0358] **90** End-abutting detection unit [0359] **91** End-abutting steering angle range determination unit [0360] **92** Counting unit [0361] **93** Relearning unit [0362] **94** End-abutting steering angle storage unit [0363] **95** Vehicle state determination unit [0364] **96**bL, **96**bR, **96**oL, **96**oR End-abutting counter [0365] **97**bL, **97**bR, **97**oL, **97**oR Provisional end-abutting counter [0366] 98bL, 98bR, 98oL, 98oR On-center counter

Claims

- 1. A turning control device comprising: a position detection unit configured to detect a steered position of a turning mechanism of a vehicle; a terminal position learning unit configured to learn a terminal position of the turning mechanism, based on the steered position detected by the position detection unit; and a relearning determination unit configured to determine necessity of relearning of the terminal position, wherein the relearning determination unit includes: an end-abutting detection unit configured to detect occurrence of end-abutting, the end-abutting being a state in which the turning mechanism is turned to the terminal position, and acquire an end-abutting steering angle, the end-abutting steering angle being a steering angle when occurrence of end-abutting is detected; an end-abutting steering angle range determination unit configured to determine whether or not variation in the end-abutting steering angles acquired multiple times is less than or equal to a predetermined threshold value; and a relearning unit configured to, when the variation is less than or equal to the predetermined threshold value, reset the learned terminal position to an initial value.
- **2.** The turning control device according to claim 1, wherein when occurrence of end-abutting is detected both in a case of steering to a left side and in a case of steering to a right side and each of variation in the end-abutting steering angles in the case of steering to the left side and variation in the end-abutting steering angles in the case of steering to the right side is less than or equal to the predetermined threshold value, the relearning unit calculates estimated rack stroke, based on the

end-abutting steering angles and resets the initial value, based on the estimated rack stroke.

- **3.** The turning control device according to claim 1 comprising a counting unit configured to count a number of times of acquisition of the end-abutting steering angles having variation less than or equal to the predetermined threshold value, wherein when the number of times of acquisition is greater than or equal to a first threshold number of times, the relearning unit resets the learned terminal position to the initial value.
- **4.** The turning control device according to claim 3, wherein the steering angle range determination unit stores the acquired end-abutting steering angle and determines whether or not variation in the end-abutting steering angles is less than or equal to the predetermined threshold value, based on a difference between the stored end-abutting steering angle and a steering angle when occurrence of end-abutting is detected, and the counting unit resets counting of the number of times of acquisition when variation in the end-abutting steering angles is determined not to be less than or equal to the predetermined threshold value.
- **5.** The turning control device according to claim 4, wherein the counting unit counts a number of times that it is determined that a second steering angle, the second steering angle being a steering angle when occurrence of end-abutting is detected, is closer to a neutral position of the turning mechanism than a first steering angle, the first steering angle being a steering angle located farthest from the neutral position among the stored end-abutting steering angles, and a difference between the first steering angle and the second steering angle is not less than or equal to the predetermined threshold value, as a number of times of being on-center, and when the number of times of being on-center is greater than or equal to a second threshold number of times, resets counting of the number of times of acquisition.
- **6.** The turning control device according to claim 3, wherein the counting unit includes a count storage unit configured to retain a number of times that the end-abutting steering angle having variation less than or equal to the predetermined threshold value is acquired during a period when vehicle speed of a vehicle is greater than or equal to a vehicle speed threshold value, as a counted number of times of end-abutting and retain a number of times that the end-abutting steering angle having variation less than or equal to the predetermined threshold value is acquired during a period when the vehicle speed is less than a vehicle speed threshold value, as a provisional counted number of times, and when the vehicle speed changes from less than the vehicle speed threshold value to greater than or equal to the vehicle speed threshold value, assigns a value of the provisional counted number of times to the counted number of times of end-abutting, and when the counted number of times of end-abutting is greater than or equal to the first threshold number of times, the relearning unit resets the learned terminal position to an initial value.
- 7. The turning control device according to claim 1, wherein the end-abutting detection unit when no end-abutting occurs in a case of steering to one of the left side and the right side and end-abutting occurs in a case of steering to the other of the left side and the right side, calculates a rack stroke, based on the learned terminal position learned on the one of the left side and the right side and the end-abutting steering angle acquired on the other of the left side and the right side, and when error of the calculated rack stroke does not fall within a predetermined allowable range, does not use the end-abutting steering angle acquired on the other of the left side and the right side for calculation of variation in the end-abutting steering angles.
- **8.** The turning control device according to claim 6, wherein the end-abutting detection unit, when end-abutting occurs both in a case of steering to the left side and in a case of steering to the right side, calculates rack stroke, based on the end-abutting steering angles acquired in the case of steering to the left side and in the case of steering to the right side, and when the calculated rack stroke is less than a predetermined lower limit, assigns a value of the counted number of end-abutting to the provisional counted number of times and resets a value of the counted number of times of end-abutting to 0.
- 9. The turning control device according to claim 1 comprising: a command value calculation unit

configured to calculate, based on an operation acting on a steering operation unit of a vehicle, a current command value for an actuator providing the turning mechanism with steering assist force; a command value correction unit configured to, when a steered position detected by the position detection unit is in a vicinity of the terminal position learned by the terminal position learning unit, correct the current command value calculated by the command value calculation unit; and a driving unit configured to drive-control the actuator, based on the current command value corrected by the command value correction unit.

10. A turning device comprising: the turning control device according to claim **9**; and an actuator configured to, drive-controlled by the turning control device, turn a steered wheel of the vehicle.