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IWATANI et al.(10) **Pub. No.: US 2025/0263116 A1**(43) **Pub. Date: Aug. 21, 2025**(54) **TURNING CONTROL DEVICE AND
TURNING DEVICE****Publication Classification**(71) Applicant: **NSK STEERING & CONTROL,
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(2013.01); **B62D 15/021** (2013.01)

(57)

ABSTRACT

A turning control device includes: a terminal position learning unit configured to learn a terminal position of the turning mechanism, based on a steered position of a turning mechanism detected by the position detection unit; and a relearning determination unit configured to determine necessity of relearning of the terminal position. The relearning determination unit includes: an end-abutting detection unit configured to detect occurrence of end-abutting, and acquire an end-abutting steering angle that is a steering angle when end-abutting is occurred; an end-abutting steering angle range determination unit configured to determine whether or not variation in the end-abutting steering angles acquired multiple times is less than or equal to a predetermined threshold value; and a relearning unit configured to, when the variation is less than or equal to the predetermined threshold value, reset the learned terminal position to an initial value.

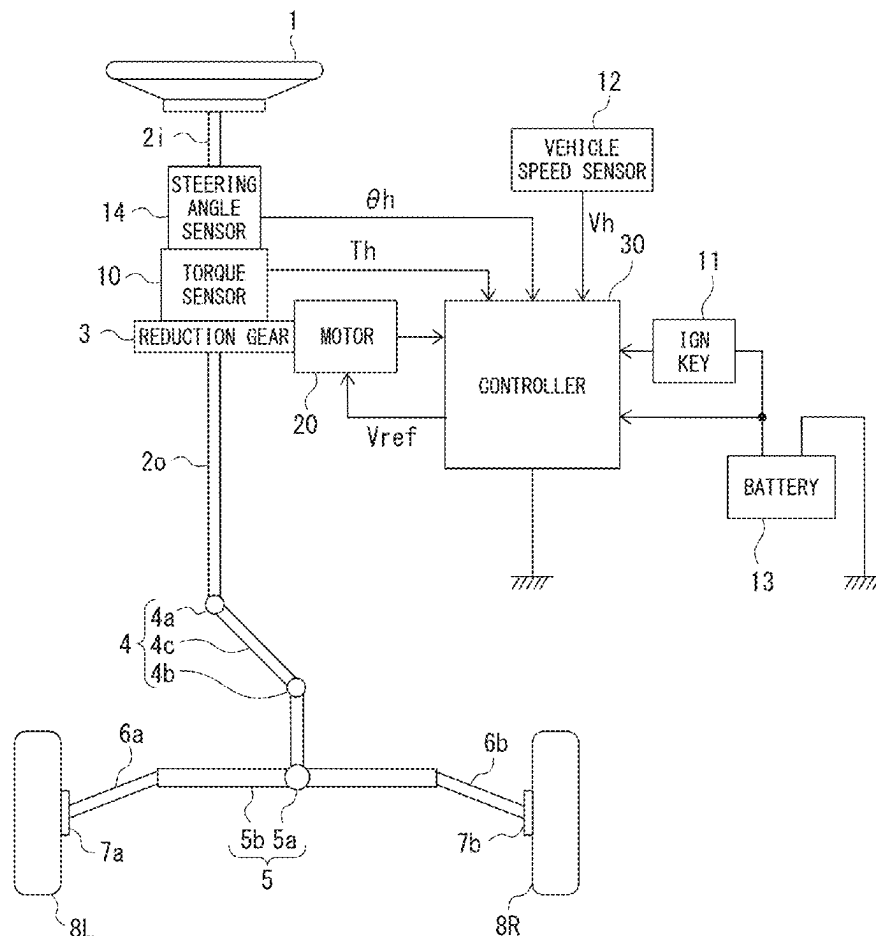


FIG. 1

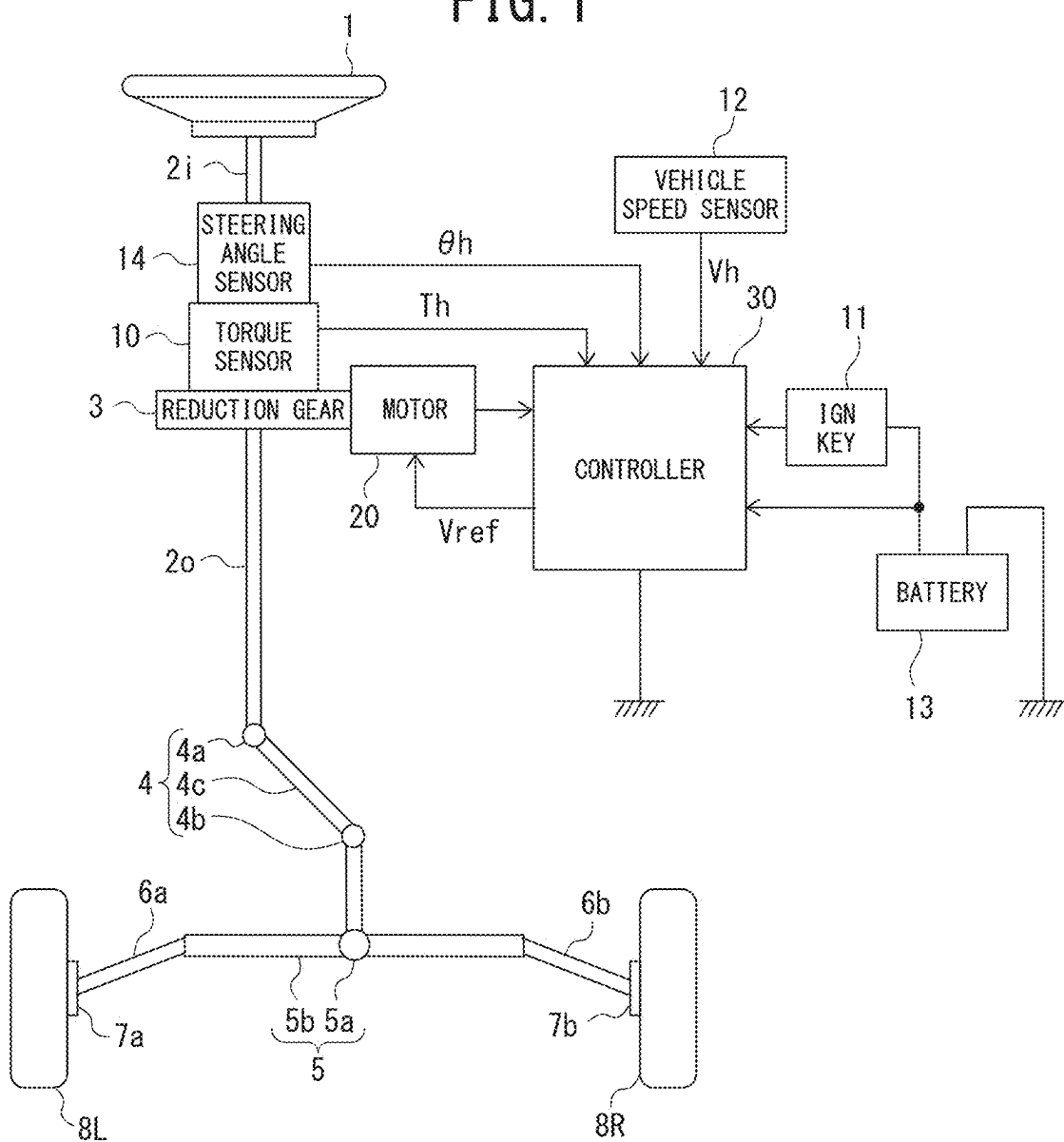


FIG. 2

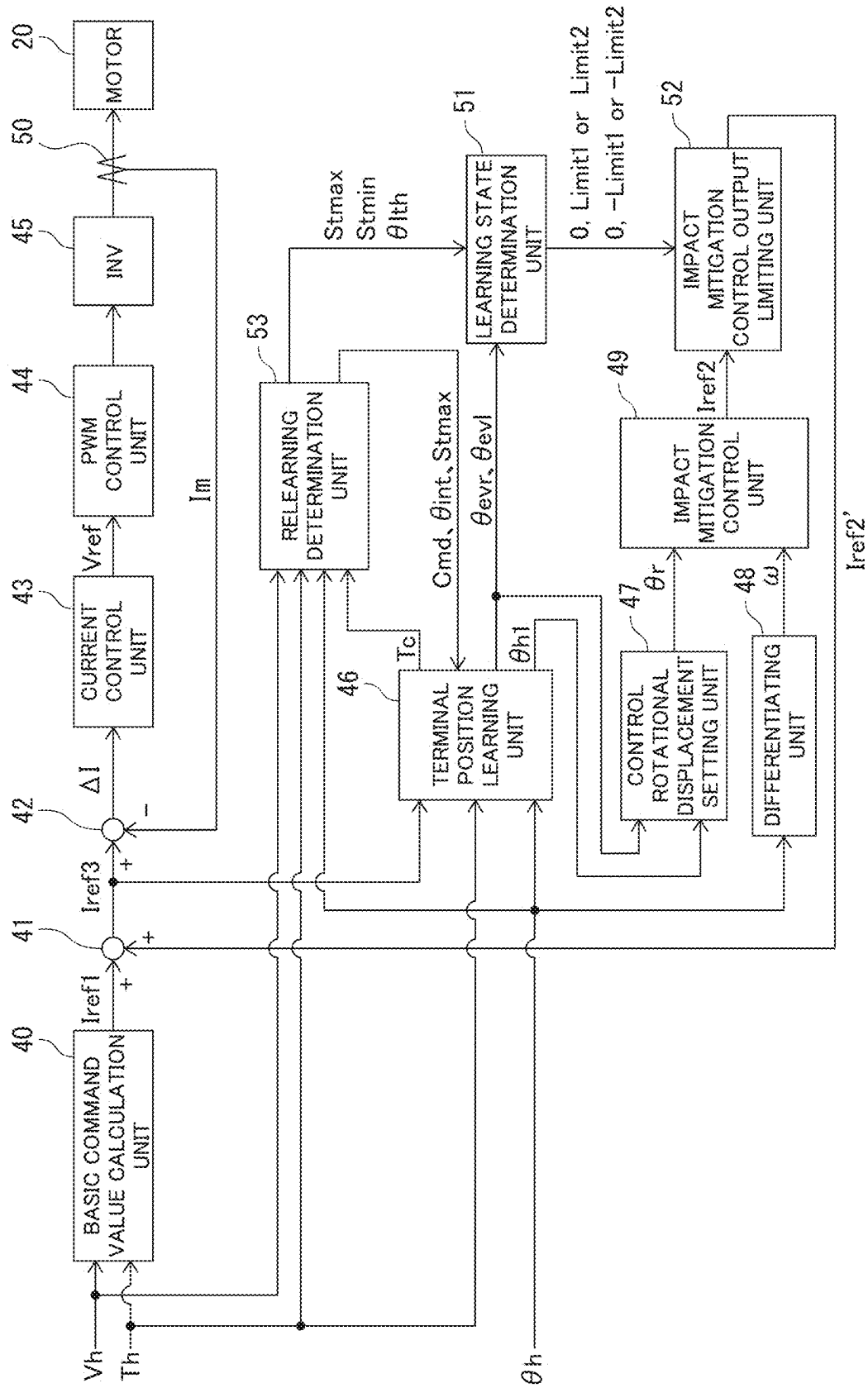


FIG. 3

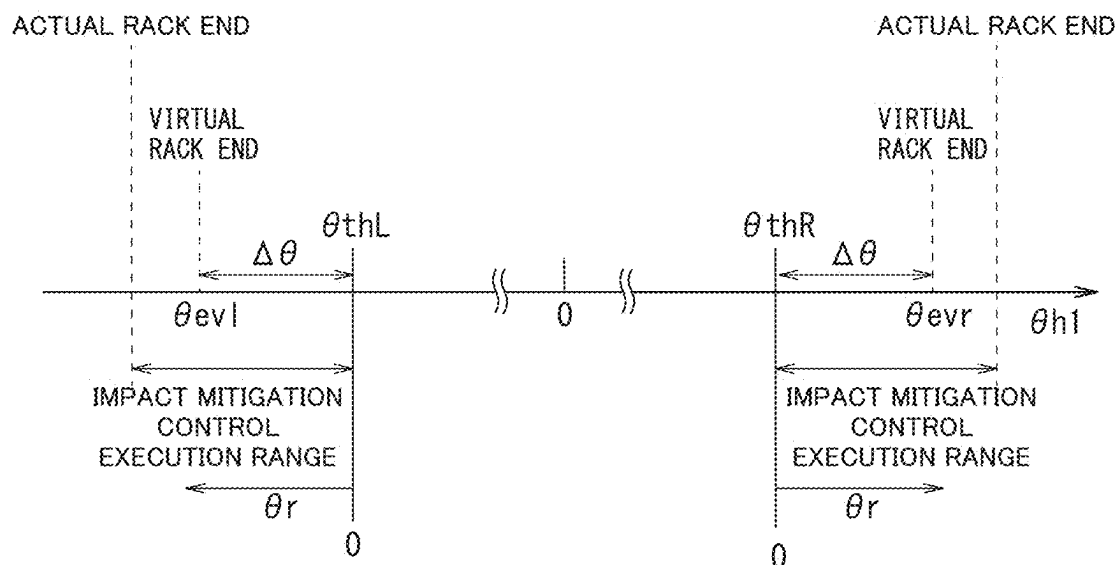


FIG. 4

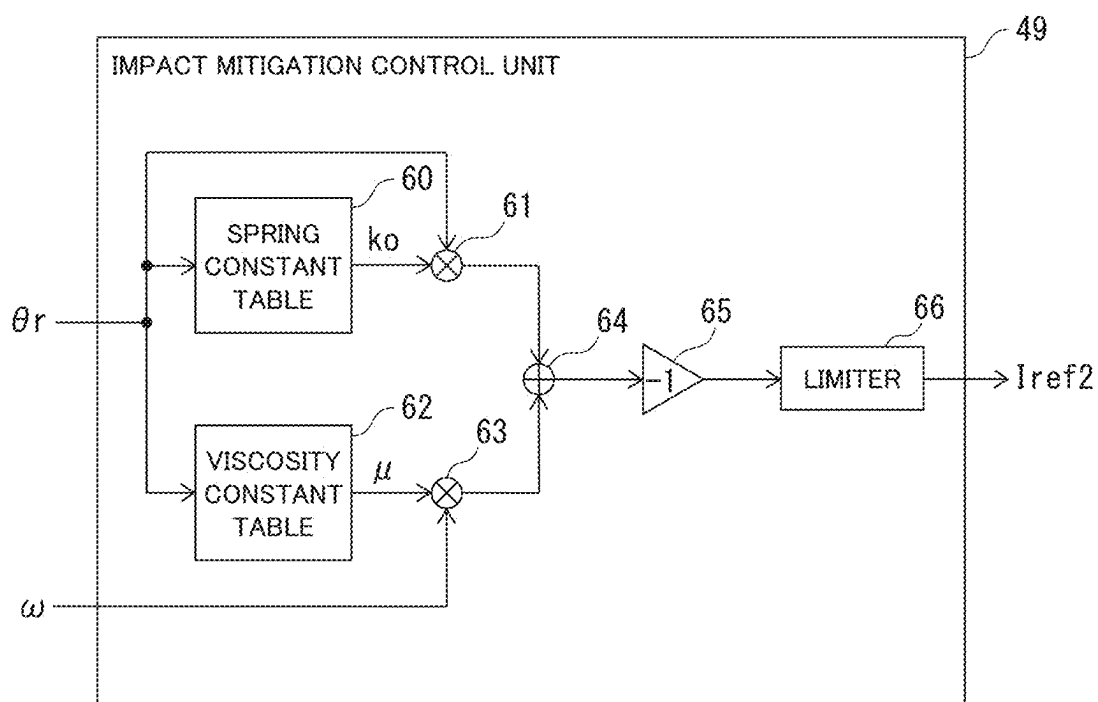


FIG. 5A

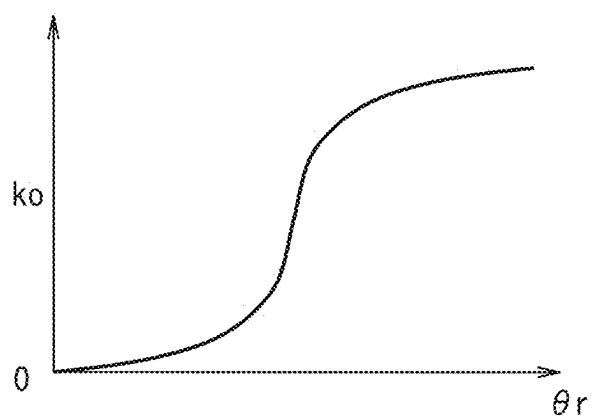


FIG. 5B

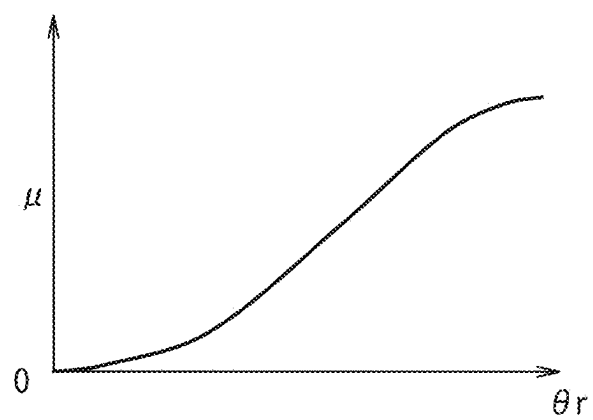


FIG. 6

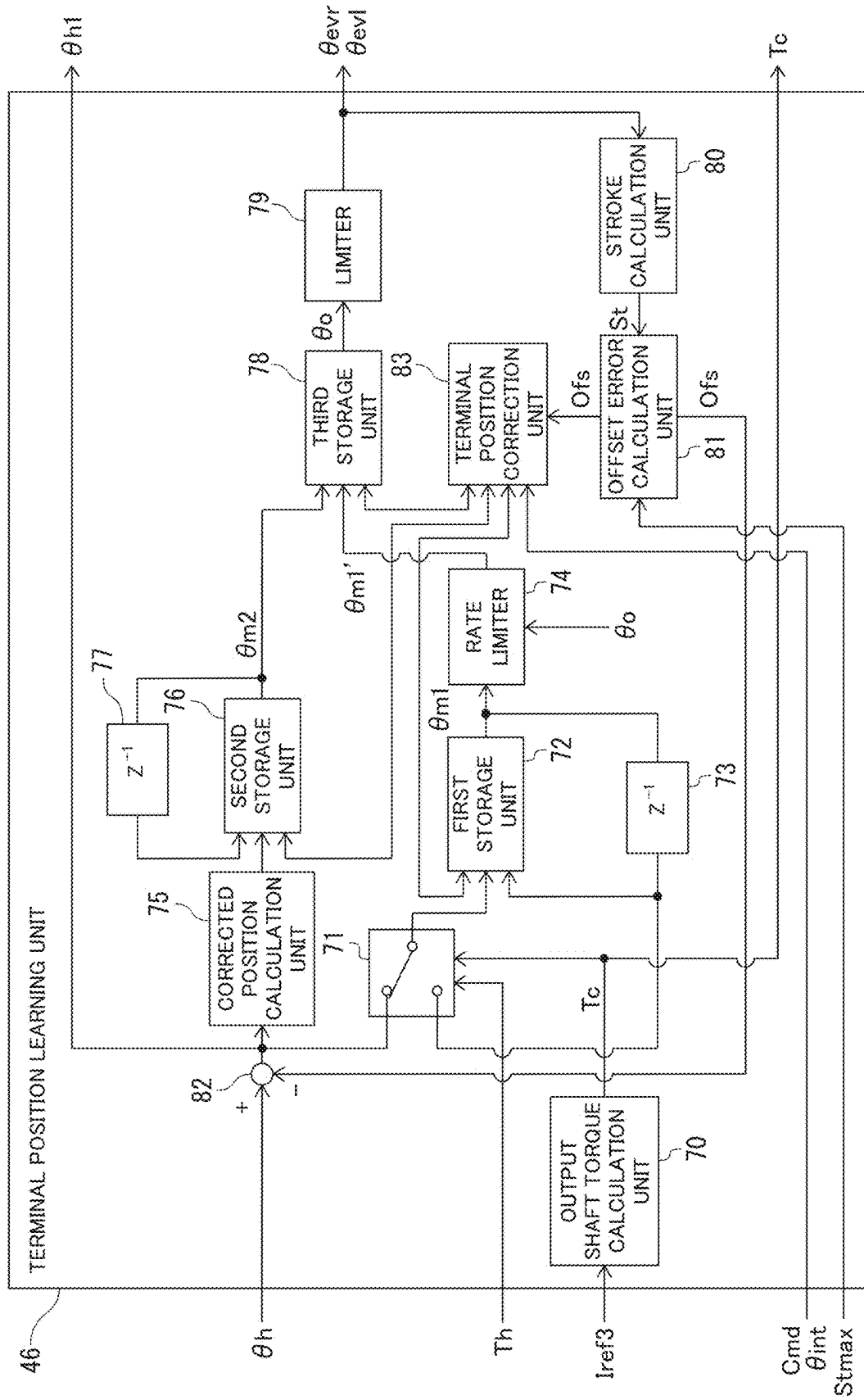


FIG. 7A

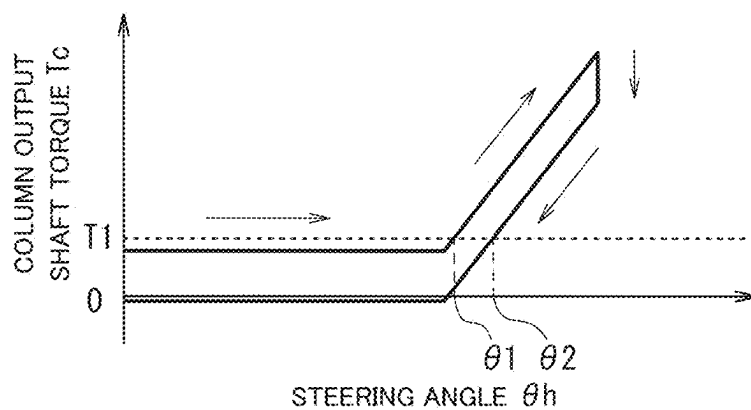


FIG. 7B

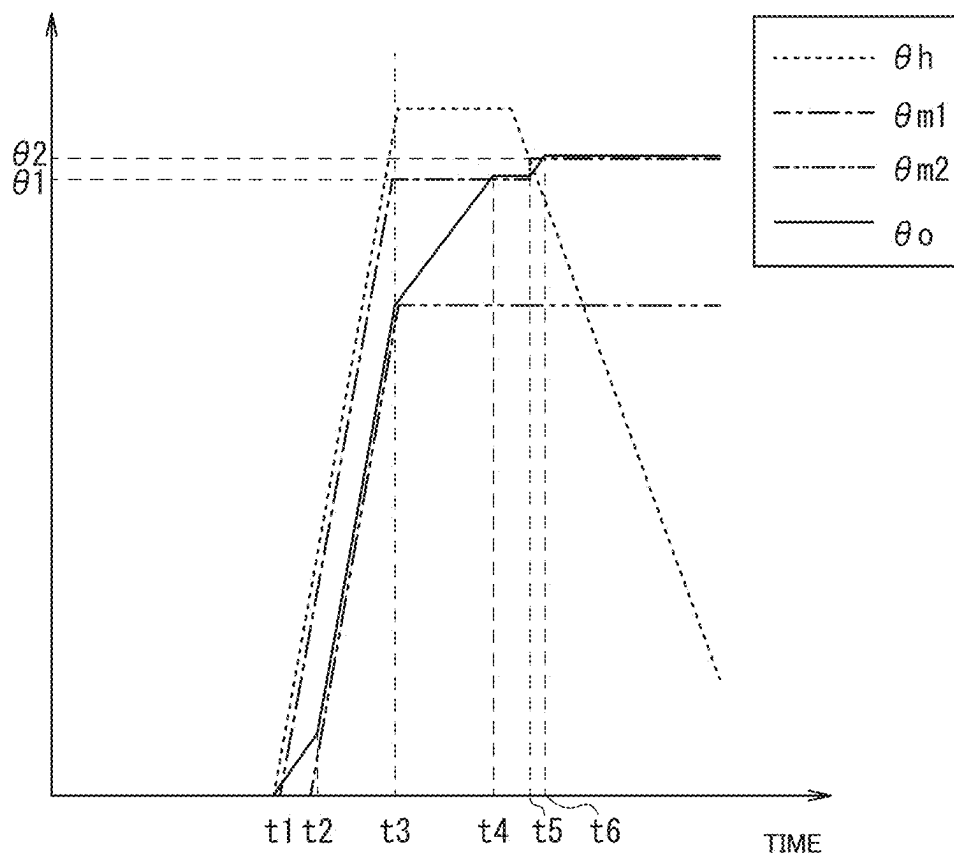


FIG. 8A

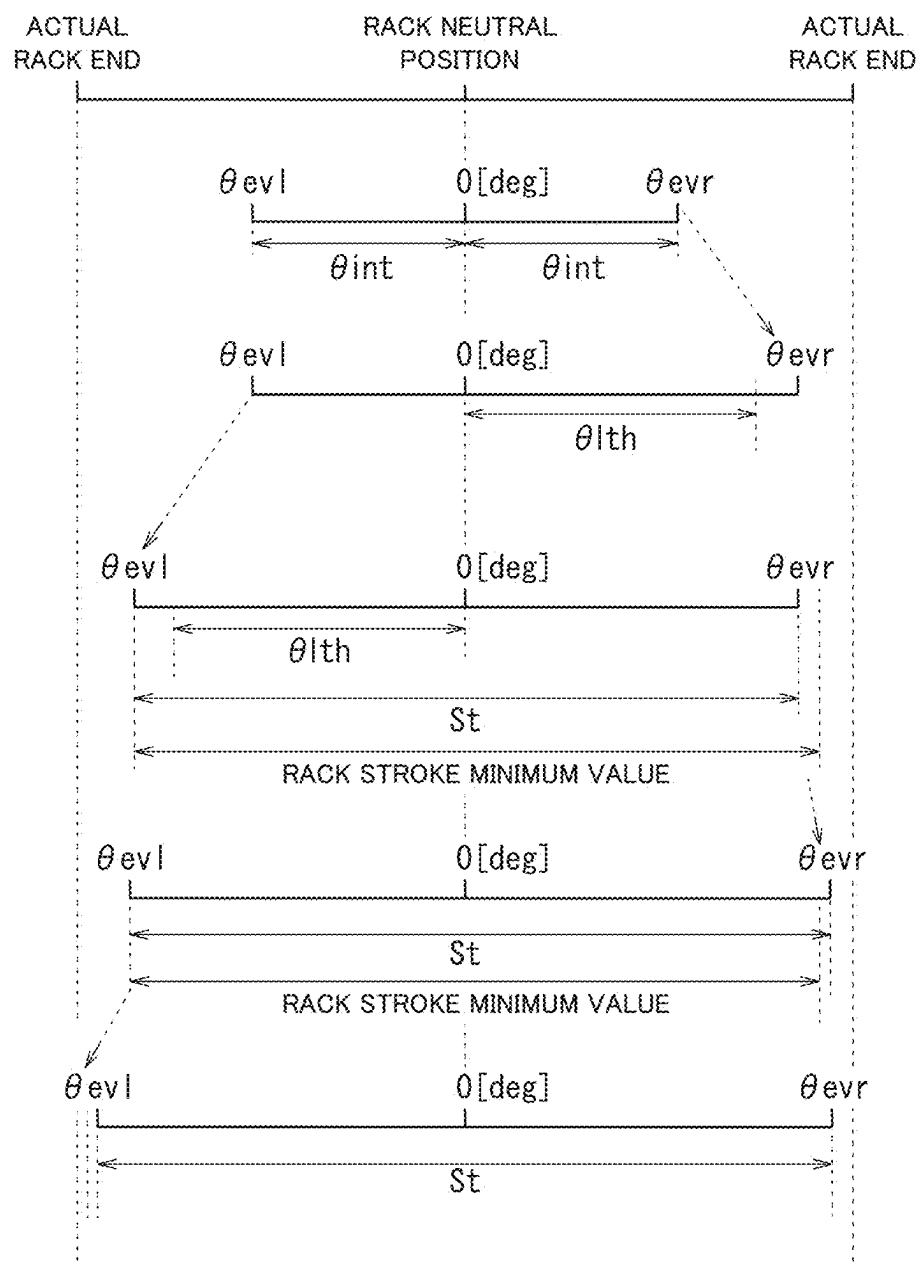
FIG. 8B

FIG. 8C

FIG. 8D

FIG. 8E

FIG. 8F



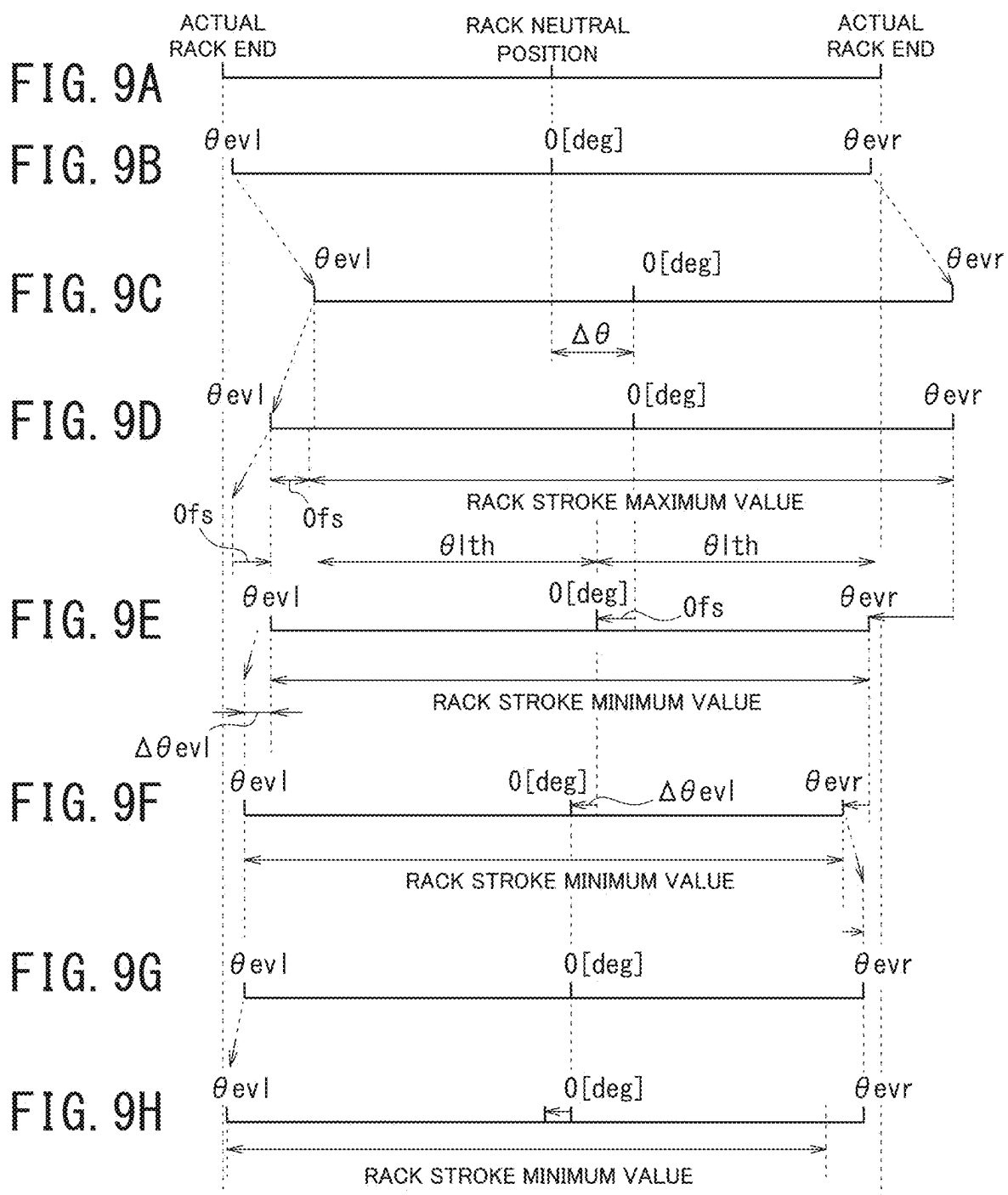


FIG. 10

STATE	UPPER LIMIT	LOWER LIMIT
FIG. 8B	0	0
FIG. 8C	0	−Limit1
FIG. 8D	Limit1	−Limit1
FIG. 8E	Limit2	−Limit2
FIG. 8F, FIG. 9B	Limit2	−Limit2
FIG. 9C	Limit2	−Limit2
FIG. 9D	0	0
FIG. 9E	Limit1	0
FIG. 9F	Limit1	0
FIG. 9G	Limit2	−Limit2
FIG. 9H	Limit2	−Limit2

FIG. 11

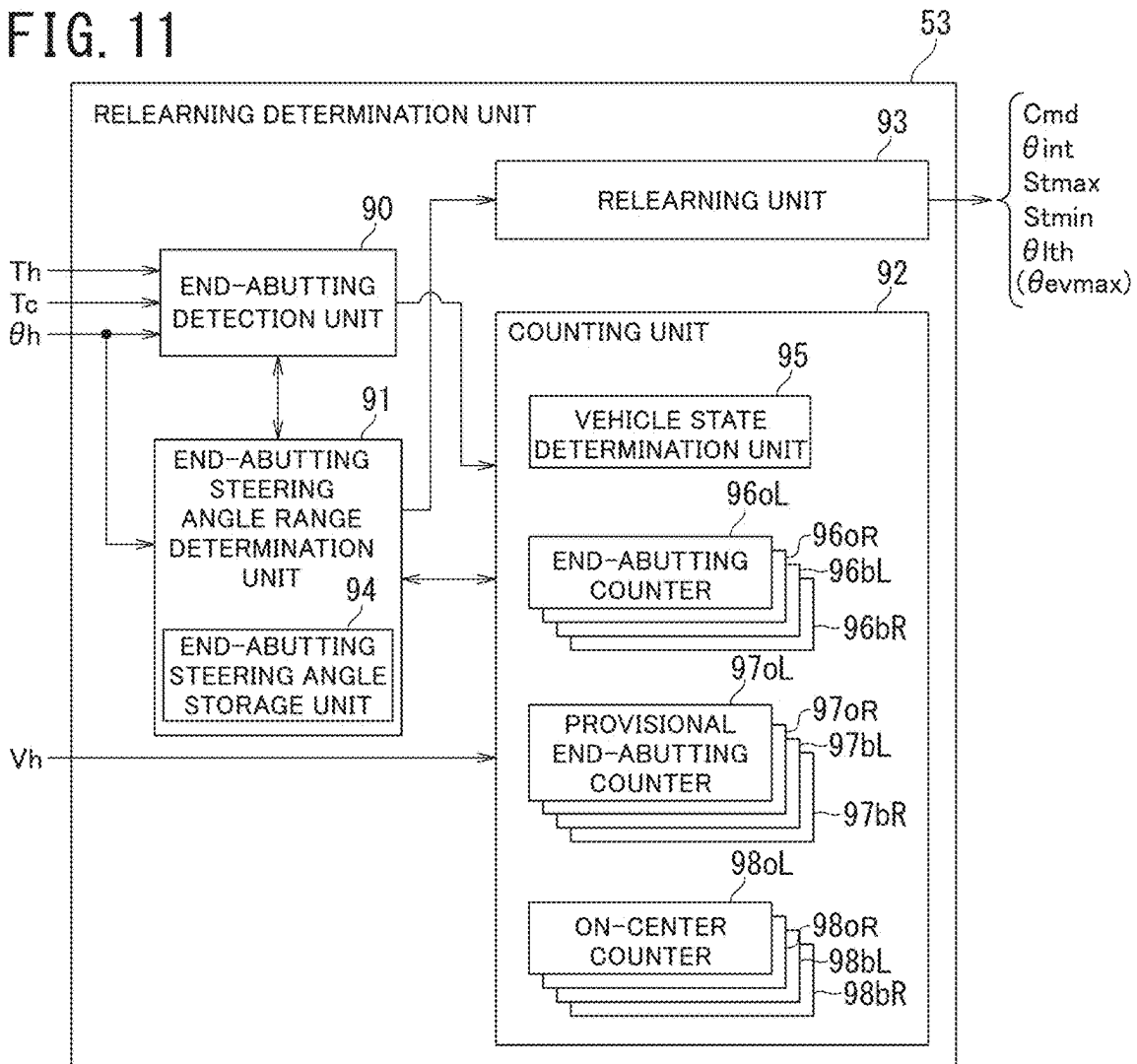


FIG. 12A

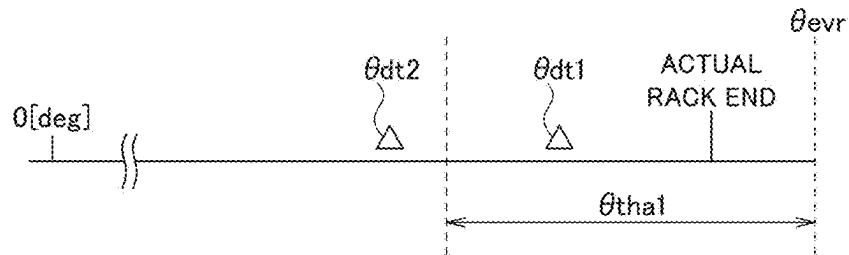


FIG. 12B

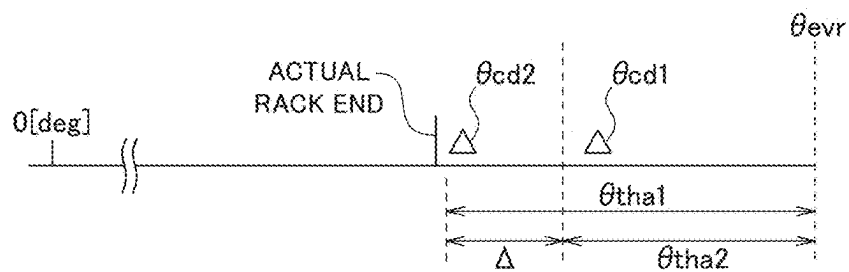


FIG. 13A

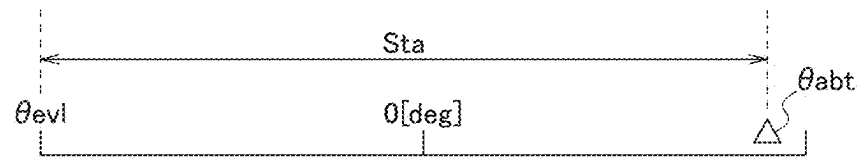


FIG. 13B

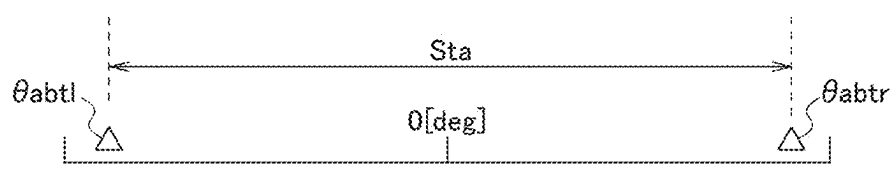


FIG. 14A

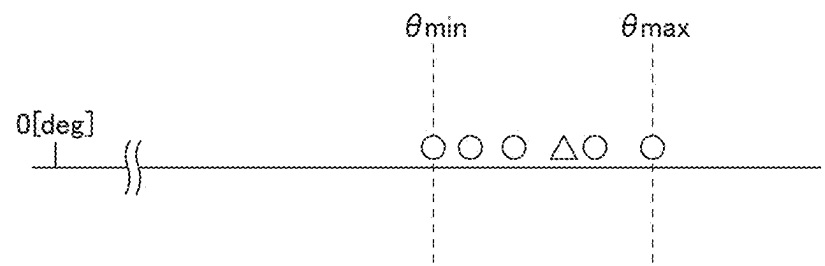


FIG. 14B

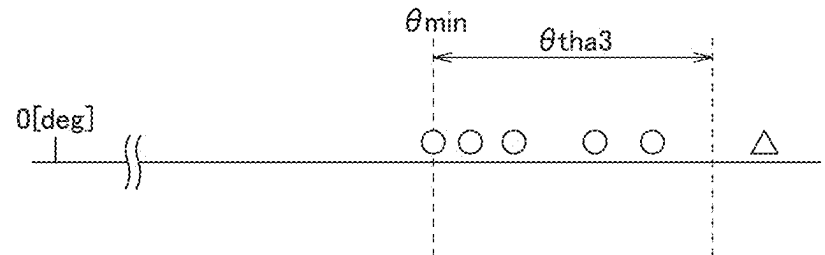


FIG. 14C

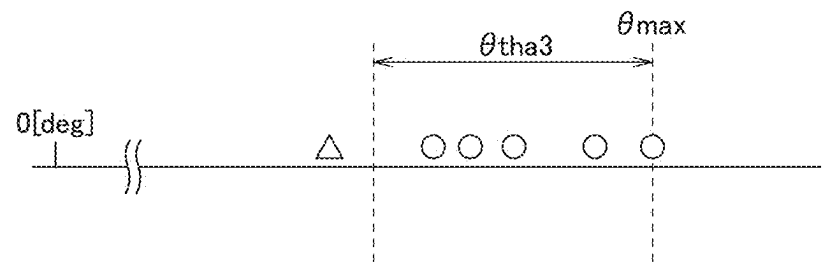


FIG. 15

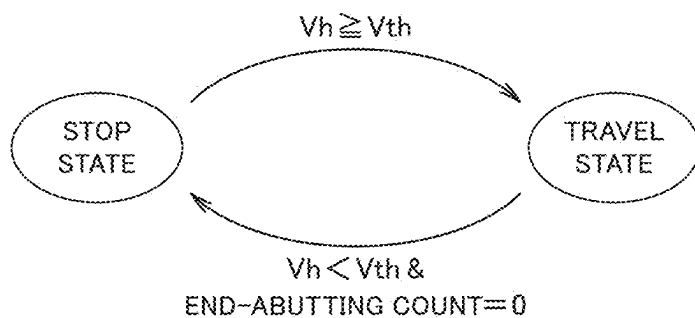


FIG. 16

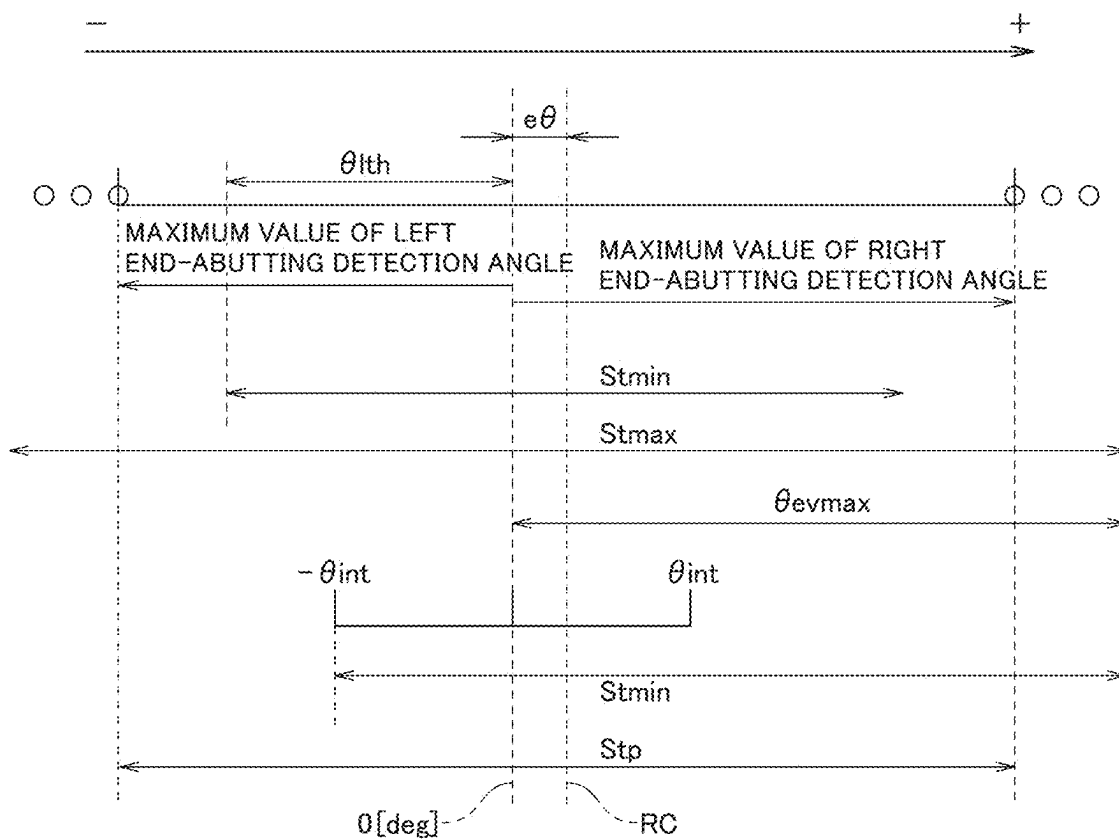


FIG. 17

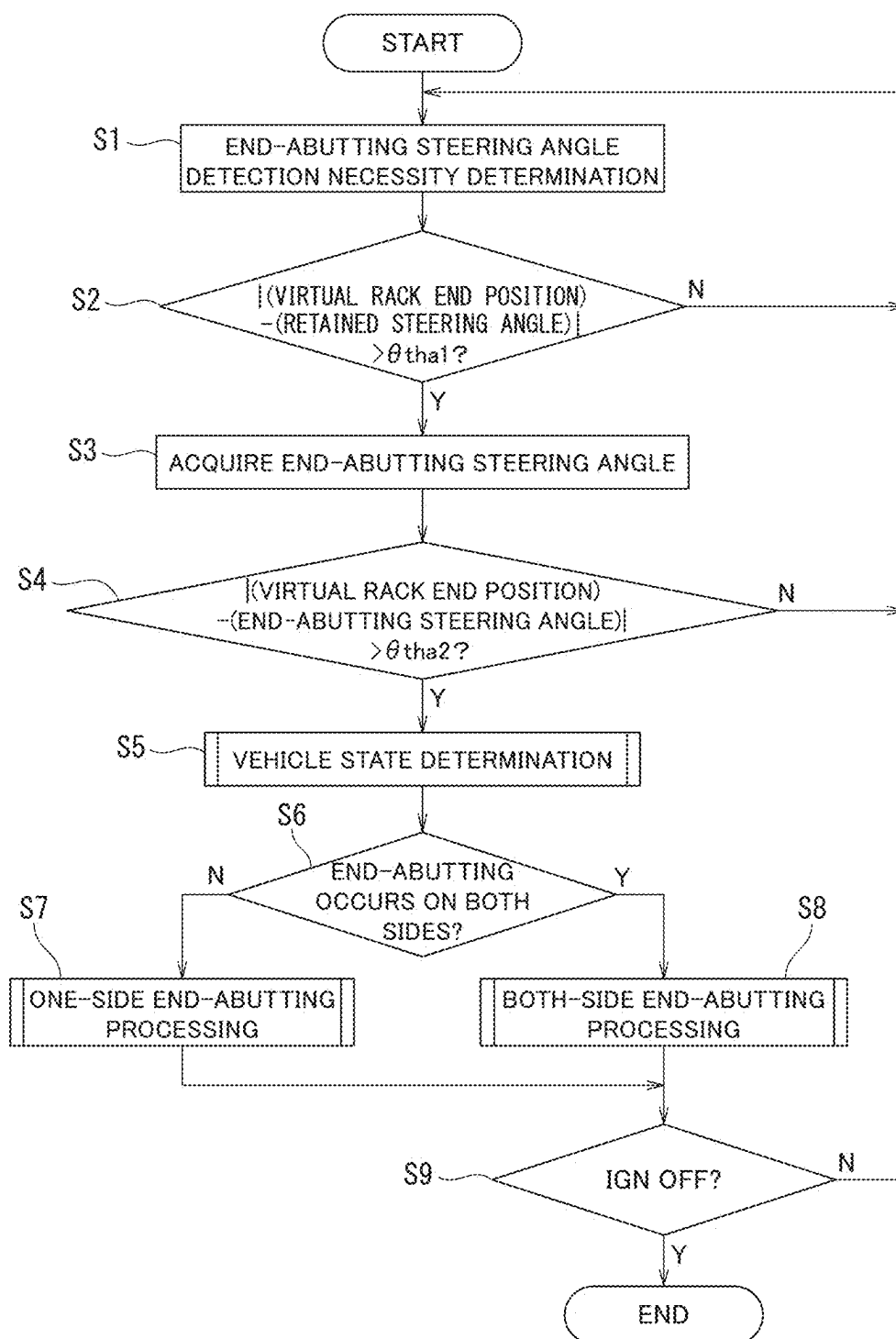


FIG. 18

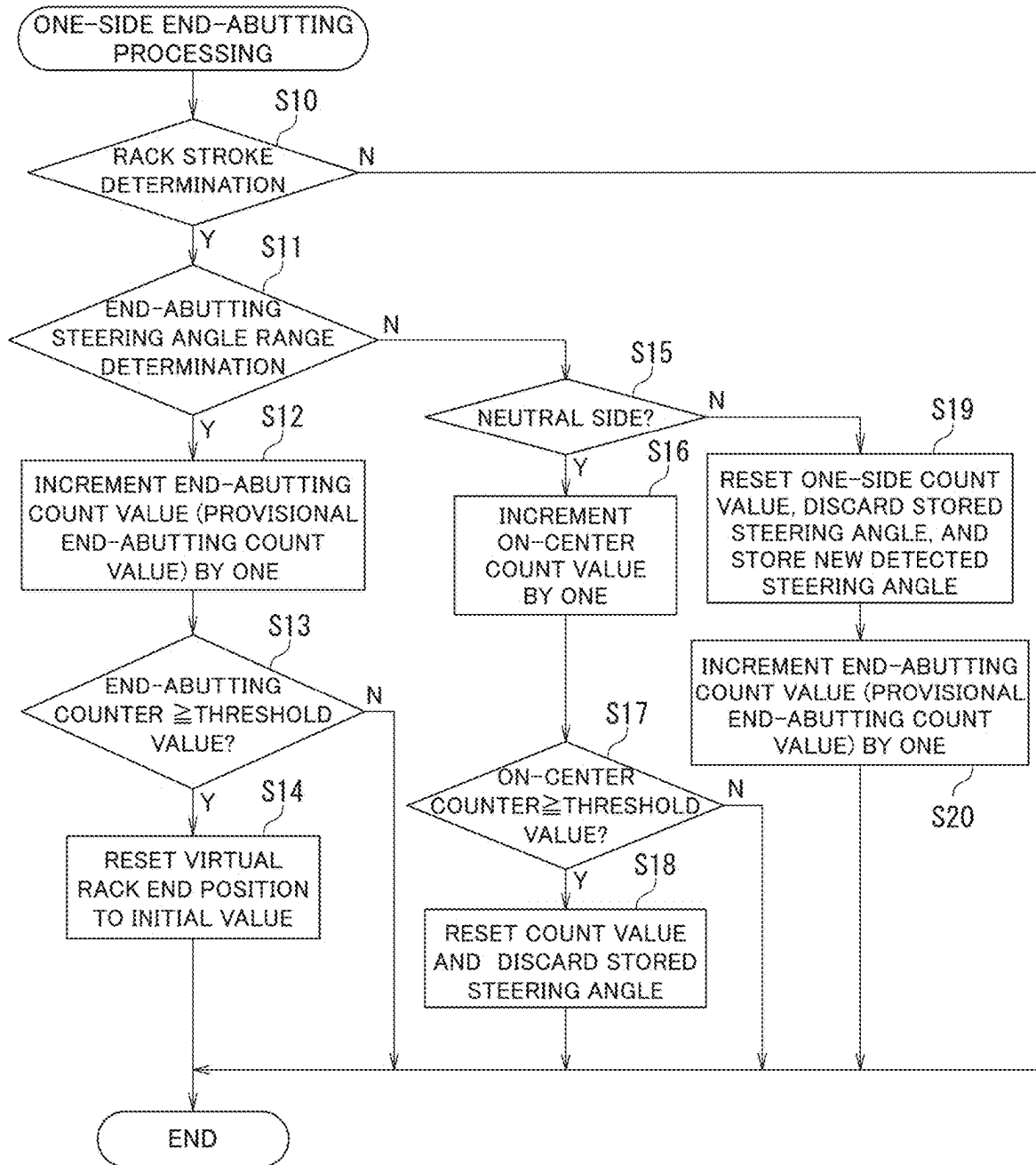
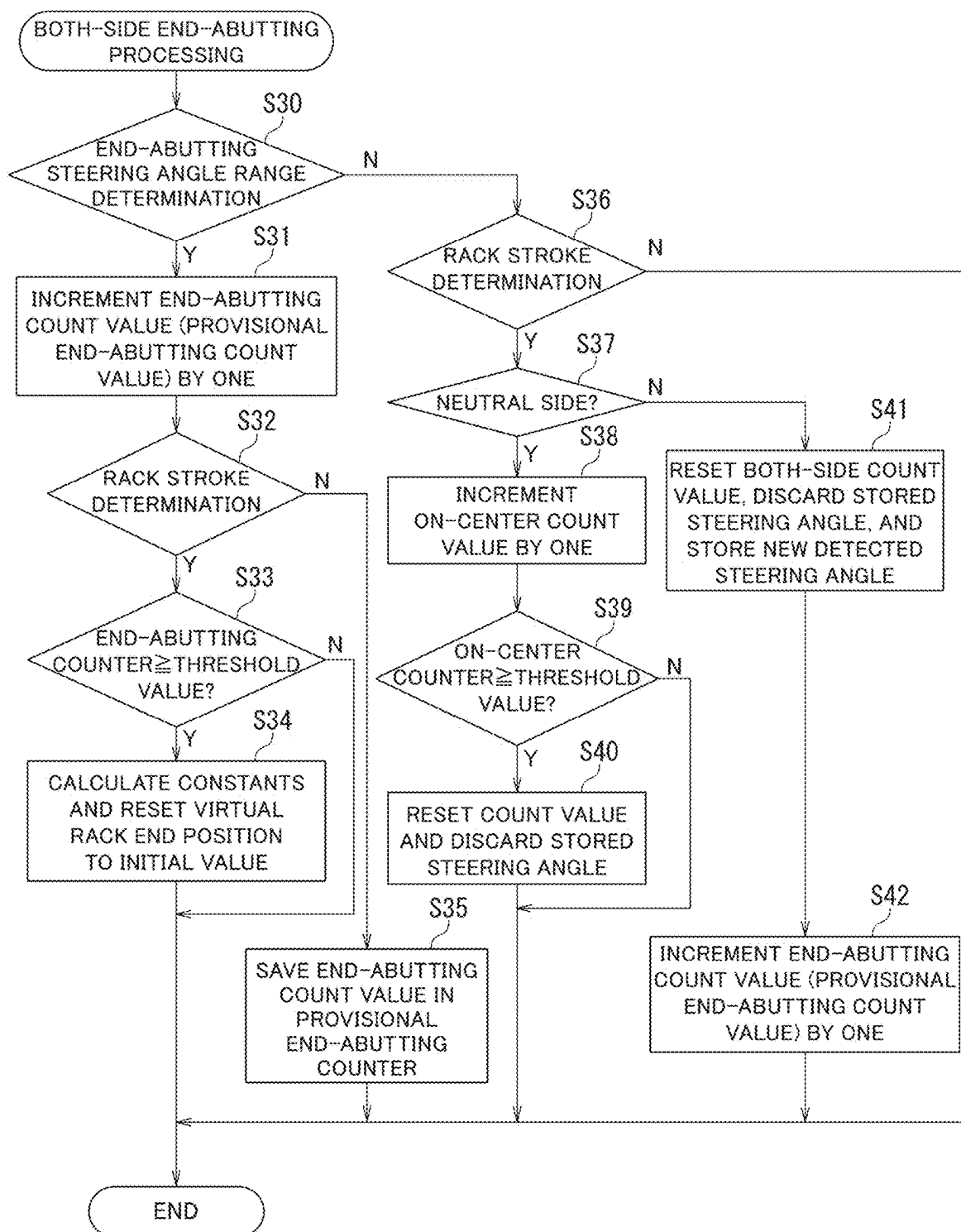


FIG. 19



TURNING CONTROL DEVICE AND TURNING DEVICE

CROSS REFERENCE TO RELATED APPLICATIONS

[0001] This application is a National Stage of International Application No. PCT/JP2024/021687 filed Jun. 14, 2024, claiming priority based on Japanese Patent Application No. 2023-121872 filed Jul. 26, 2023.

TECHNICAL FIELD

[0002] The present invention relates to a turning control device and a turning device.

BACKGROUND ART

[0003] In a turning mechanism of a vehicle, when a turning angle increases and reaches a maximum mechanical turning angle, a rack shaft of the turning mechanism reaches a stroke end and it becomes impossible to increase the turning angle any further. The rack shaft being brought to a state of having reached a stroke end as described above is referred to as “end-abutting”. In addition, a stroke end of the rack shaft is sometimes referred to as “rack end”.

[0004] When end-abutting occurs at a high turning velocity, there is a risk that large impact and hit sound (abnormal noise) are generated and the driver feels uncomfortable. In PTLs 1 and 2, technologies for mitigating impact at the time of end-abutting by learning a maximum value of an absolute value of a steering angle as rack end positions and preventing increase in a steering angle when the steering angle detected by a sensor is in a vicinity of one of the learned rack end positions are described.

[0005] Hereinafter, a virtual rack end position learned based on a steering angle detected by the sensor is sometimes referred to as “virtual rack end position”, and an actual physical rack end position is sometimes referred to as “actual rack end position”.

CITATION LIST

Patent Literature

[0006] PTL 1: JP 7131737 B

[0007] PTL 2: JP 7136398 B

SUMMARY OF INVENTION

Technical Problem

[0008] There are some cases where after virtual rack end positions are learned, a rack shaft is replaced. On this occasion, it is conceivable that a rack shaft having a length different from length of a rack shaft that should be originally mounted is mistakenly mounted or a mounting position of a rack shaft changes between before and after replacement of the rack shaft. In such a case, it is necessary to relearn the virtual rack end positions. There is a risk that when the virtual rack end positions are relearned, an inappropriate virtual rack end position is learned due to a reason such as rapid turning-back steering and collision of a tire with a curb.

[0009] The present invention has been made in consideration of the above-described problem, and an object of the

present invention is to prevent incorrect learning at the time of relearning of a rack end position.

Solution to Problem

[0010] In order to achieve the above-described object, according to an aspect of the present invention, there is provided a turning control device including: a position detection unit configured to detect a steered position of a turning mechanism of a vehicle; a terminal position learning unit configured to learn a terminal position of the turning mechanism, based on the steered position detected by the position detection unit; and a relearning determination unit configured to determine necessity of relearning of the terminal position, wherein the relearning determination unit includes: an end-abutting detection unit configured to detect occurrence of end-abutting, the end-abutting being a state in which the turning mechanism is turned to the terminal position, and acquire an end-abutting steering angle, the end-abutting steering angle being a steering angle when occurrence of end-abutting is detected; an end-abutting steering angle range determination unit configured to determine whether or not variation in the end-abutting steering angles acquired multiple times is less than or equal to a predetermined threshold value; and a relearning unit configured to, when the variation is less than or equal to the predetermined threshold value, reset the learned terminal position to an initial value.

[0011] According to another aspect of the present invention, there is provided a turning device including: the turning control device described above; and an actuator configured to, drive-controlled by the turning control device, turn a steered wheel of the vehicle.

Advantageous Effects of Invention

[0012] According to the present invention, it is possible to prevent incorrect learning at the time of relearning of a rack end position.

BRIEF DESCRIPTION OF DRAWINGS

[0013] FIG. 1 is a configuration diagram illustrative of an outline of an example of an electric power steering device of an embodiment;

[0014] FIG. 2 is a block diagram illustrative of an example of a functional configuration of a controller illustrated in FIG. 1;

[0015] FIG. 3 is an explanatory diagram of an example of a steering angle range where impact mitigation control is performed;

[0016] FIG. 4 is a block diagram illustrative of an example of a functional configuration of an impact mitigation control unit;

[0017] FIGS. 5A and 5B are a characteristic diagram illustrative of a characteristic example of a spring constant table and a characteristic diagram illustrative of a characteristic example of a viscosity constant table, respectively;

[0018] FIG. 6 is a block diagram illustrative of an example of a functional configuration of a terminal position learning unit;

[0019] FIGS. 7A and 7B are an explanatory diagram of an example of change in column output shaft torque associated with change in a steering angle and an explanatory diagram

of an example of a learned value of a terminal position when the column output shaft torque in FIG. 7A is generated, respectively;

[0020] FIGS. 8A, 8B, 8C, 8D, 8E, and 8F are a conceptual diagram of actual rack end positions, a conceptual diagram of a state before start of learning of virtual rack end positions, a conceptual diagram of a state in which a right virtual rack end position has been learned, a conceptual diagram of a state in which a left virtual rack end position has been learned, a conceptual diagram of a state in which learning of the virtual rack ends is considered to have been completed, and a conceptual diagram of a state in which the learning of the virtual rack ends has been continued to the vicinities of the actual rack ends, respectively;

[0021] FIGS. 9A, 9B, 9C, 9D, 9E, 9F, 9G, and 9H are a conceptual diagram of the actual rack end positions, a conceptual diagram illustrative of the same state as the state in FIG. 8F, a conceptual diagram of a state immediately after offset error has occurred, a conceptual diagram of a state in which a learned position of the left virtual rack end position is updated, a conceptual diagram of correction of a steering angle detected by a steering angle sensor and resetting of a learned value of the right virtual rack end position, a conceptual diagram of a state in which a learned position of the left virtual rack end position is further updated, a conceptual diagram of a state in which learning of the virtual rack ends is considered to have been completed, and a conceptual diagram of a state in which a learned position of the left virtual rack end position is further updated, respectively;

[0022] FIG. 10 is a table illustrative of an example of limiting values in the states in FIGS. 8B to 8F and FIGS. 9B to 9H;

[0023] FIG. 11 is a block diagram illustrative of an example of a functional configuration of a relearning determination unit;

[0024] FIGS. 12A and 12B are an explanatory diagram of an example of end-abutting steering angle necessity determination and an explanatory diagram of an example of end-abutting steering angle acquisition possibility determination, respectively;

[0025] FIGS. 13A and 13B are explanatory diagrams of examples of rack stroke determination;

[0026] FIGS. 14A to 14C are explanatory diagrams of examples of end-abutting steering angle range determination;

[0027] FIG. 15 is an example of a state transition diagram of a vehicle state;

[0028] FIG. 16 is an explanatory diagram of an example of a method for calculating a rack stroke minimum value St_{min} , a rack stroke maximum value St_{max} , a learning threshold value θ_{lth} , a rack end maximum value θ_{evmax} , and an initial value θ_{int} ;

[0029] FIG. 17 is a flowchart of an example of a turning control method of the embodiment;

[0030] FIG. 18 is a flowchart of an example of one-side end-abutting processing in FIG. 17; and

[0031] FIG. 19 is a flowchart of an example of both-side end-abutting processing in FIG. 17.

DESCRIPTION OF EMBODIMENTS

[0032] Embodiments of the present invention will be described in detail with reference to the drawings. Note that the embodiments of the present invention to be described

below indicate devices and methods to embody the technical idea of the present invention by way of example, and the technical idea of the present invention does not limit the constitution, arrangements, and the like of the constituent components to those described below. The technical idea of the present invention can be subjected to a variety of alterations within the technical scope prescribed by the claims described in CLAIMS.

(Configuration)

[0033] FIG. 1 is a configuration diagram illustrative of an outline of an example of an electric power steering device of an embodiment. Column shafts (steering shafts) $2i$ and $2o$ of a steering wheel 1 is connected to steered wheels $8L$ and $8R$ by way of a reduction gear (worm gear) 3, which constitutes a speed reduction mechanism, an intermediate shaft 4, a pinion rack mechanism 5, and tie rods $6a$ and $6b$ and further via hub units $7a$ and $7b$.

[0034] The column input shaft $2i$ and the column output shaft $2o$ are connected by a torsion bar (not illustrated) that is twisted due to a difference in rotation angles between the column input shaft $2i$ and the column output shaft $2o$.

[0035] The intermediate shaft 4 includes a shaft member $4c$ and universal joints $4a$ and $4b$ that are attached to both ends of the shaft member. The universal joint $4a$ is coupled to the column output shaft $2o$, and the universal joint $4b$ is coupled to the pinion rack mechanism 5.

[0036] The pinion rack mechanism 5 includes a pinion $5a$ that is coupled to a pinion shaft to which steering force is transmitted from the universal joint $4b$ and a rack $5b$ that meshes with the pinion $5a$, and converts rotational motion transmitted to the pinion $5a$ to linear motion in the vehicle width direction by the rack $5b$.

[0037] To the steering shaft 2 (column shafts $2i$ and $2o$), a torque sensor 10 configured to detect steering torque Th is disposed. To the steering shaft 2 (column shafts $2i$ and $2o$), a steering angle sensor 14 configured to detect a steering angle θ_h of the steering wheel 1 is also disposed.

[0038] A motor 20 configured to assist steering force of the steering wheel 1 is also connected to the column output shaft $2o$ via the reduction gear 3. To a controller 30 configured to control the electric power steering (EPS) device, power is supplied from a battery 13 and an ignition key signal is also input via an ignition (IGN) key 11.

[0039] Note that a means for providing steering assist force is not limited to a motor and a variety of types of actuators can be used.

[0040] The controller 30 performs calculation of a current command value of an assist control command, based on steering torque Th detected by the torque sensor 10, vehicle speed V_h detected by a vehicle speed sensor 12, and a steering angle θ_h detected by the steering angle sensor 14 and controls current to be supplied to the motor 20 by a voltage control command value V_{ref} obtained by performing compensation and the like on the calculated current command value.

[0041] Note that the steering angle sensor 14 is not an essential component and the steering angle θ_h may be calculated by adding a torsion angle of a torsion bar in the torque sensor 10 to a rotation angle obtained from a rotation angle sensor configured to detect a rotation angle of the rotation shaft of the motor 20.

[0042] The controller 30 may include, for example, a computer including a processor and peripheral components,

such as a storage device. The processor may be, for example, a central processing unit (CPU) or a micro-processing unit (MPU).

[0043] The storage device may include any one of a semiconductor storage device, a magnetic storage device, and an optical storage device. The storage device may include registers, a cache memory, or a memory, such as a read only memory (ROM) and a random access memory (RAM), that is used as a main storage device.

[0044] Functions of the controller 30, which will be described below, are achieved by, for example, the processor of the controller 30 executing computer programs stored in the storage device.

[0045] Note that the controller 30 may be formed by use of dedicated hardware for performing respective units of information processing, which will be described below. For example, the controller 30 may include functional logic circuits that are set in a general-purpose semiconductor integrated circuit. For example, the controller 30 may have a programmable logic device (PLD), such as a field-programmable gate array (FPGA), or the like.

[0046] FIG. 2 is a block diagram illustrative of an example of a functional configuration of the controller 30 of the embodiment. The controller 30 includes a basic command value calculation unit 40, an adder 41, a subtracter 42, a current control unit 43, a pulse width modulation (PWM) control unit 44, an inverter (INV) 45, a terminal position learning unit 46, a control rotational displacement setting unit 47, a differentiating unit 48, an impact mitigation control unit 49, a current detector 50, a learning state determination unit 51, an impact mitigation control output limiting unit 52, and a relearning determination unit 53.

[0047] The basic command value calculation unit 40 calculates a basic current command value Iref1 that is a control target value of driving current of the motor 20, based on the steering torque Th from the torque sensor 10 and the vehicle speed Vh from the vehicle speed sensor 12.

[0048] In the present embodiment, a value of the basic current command value Iref1 that causes a steering assist force of the motor 20 to be generated in the rightward steering direction is defined as a positive value, and a value of a basic current command value Iref1 that causes a steering assist force to be generated in the leftward steering direction is defined as a negative value.

[0049] The adder 41 corrects the basic current command value Iref1 by adding impact mitigation control output Iref2' output from the impact mitigation control output limiting unit 52 to the basic current command value Iref1 and outputs the basic current command value Iref1 after correction as a current command value Iref3.

[0050] The impact mitigation control output limiting unit 52 sets the impact mitigation control output Iref2' by limiting an upper limit of impact mitigation control output Iref2 output from the impact mitigation control unit 49 by a limiting value 0, Limit1, or Limit2 output from the learning state determination unit 51 and limiting a lower limit of the impact mitigation control output Iref2 by a limiting value 0, -Limit1, or -Limit2 output from the learning state determination unit 51.

[0051] The impact mitigation control unit 49 mitigates impact and hit sound (abnormal noise) due to end-abutting by suppressing increase in the steering angle θ_h when the steering angle θ_h comes close to a rack end position. Control performed by the impact mitigation control unit 49 to

mitigate impact and abnormal noise due to end-abutting is sometimes referred to as “impact mitigation control”.

[0052] The impact mitigation control unit 49 outputs a current command value to suppress increase in the steering angle θ_h in order to mitigate impact and hit sound due to end-abutting, as the impact mitigation control output Iref2. The impact mitigation control output Iref2 at the time of rightward steering has a negative value and reduces the magnitude of the positive basic current command value Iref1. On the other hand, the impact mitigation control output Iref2 at the time of leftward steering has a positive value and reduces the magnitude of the negative basic current command value Iref1. For example, the impact mitigation control unit 49 may output a current command value to generate a steering reaction force.

[0053] The impact mitigation control output limiting unit 52 limits the upper limit of the impact mitigation control output Iref2 at the time of leftward steering to a limiting value 0 or a positive limiting value Limit1 or Limit2 output from the learning state determination unit 51 and limits the lower limit of the impact mitigation control output Iref2 at the time of rightward steering to a limiting value 0 or a negative limiting value -Limit1 or -Limit2 output from the learning state determination unit 51.

[0054] Details of the impact mitigation control unit 49, the learning state determination unit 51, and the relearning determination unit 53 will be described later.

[0055] The current command value Iref3 that the adder 41 calculated is input to the subtracter 42, and deviation Iref3-Im of the current command value Iref3 from a fed-back motor current value Im is calculated. The deviation Iref3-Im is controlled by the current control unit 43 configured to perform PI control or the like, a current-controlled voltage control value Vref is input to the PWM control unit 44 and a duty cycle is calculated therein, and a PWM signal PWM-drives the motor 20 via the inverter 45. The motor current value Im of the motor 20 is detected by the current detector 50, and input to the subtracter 42 and thereby fed back.

[0056] The terminal position learning unit 46 learns, based on the steering angle θ_h detected by the steering angle sensor 14, virtual rack end positions θ_{evr} and θ_{evl} that are terminal positions of a turning mechanism. θ_{evr} denotes a virtual rack end position at the time of rightward steering and has a positive value. θ_{evl} denotes a virtual rack end position at the time of leftward steering and has a negative value.

[0057] Further, error sometimes occurs between a central position between right and left actual rack end positions (hereinafter, sometimes referred to as “rack neutral position”) and a neutral position of the steering angle θ_h of the column shaft detected by the steering angle sensor 14 (hereinafter, sometimes referred to as “steering angle neutral position”). Hereinafter, such error is sometime referred to as “offset error”.

[0058] The offset error occurs due to reasons such as incorrect assembly of the intermediate shaft 4. The terminal position learning unit 46 estimates offset error Of_s and outputs a corrected steering angle θ_{h1} to which the steering angle θ_h detected by the steering angle sensor 14 is corrected by subtracting the offset error Of_s from the steering angle θ_h . Details of the terminal position learning unit 46 will be described later.

[0059] The control rotational displacement setting unit 47 sets control rotational displacement θ_r that indicates how close the corrected steering angle θ_{h1} is to one of the virtual rack end positions θ_{evr} and θ_{evl} when the corrected steering angle θ_{h1} comes close to a rack end position and is within a range where the impact mitigation control is executed (hereinafter, sometimes referred to as “impact mitigation control execution range”).

[0060] FIG. 3 is now referred to. In the case of the rightward steering (that is, in the case where the corrected steering angle θ_{h1} is a positive value), a range where the corrected steering angle θ_{h1} is greater than a threshold value θ_{thR} is the impact mitigation control execution range, and in the case of the leftward steering (that is, in the case where the corrected steering angle θ_{h1} is a negative value), a range where the corrected steering angle θ_{h1} is less than a threshold value θ_{thL} is the impact mitigation control execution range.

[0061] The threshold values θ_{thR} and θ_{thL} are set based on the virtual rack end positions θ_{evr} and θ_{evl} , respectively. For example, the threshold value θ_{thR} for the rightward steering may be a value $\theta_{evr} - \Delta\theta$ obtained by subtracting a positive predetermined value $\Delta\theta$ from the virtual rack end position θ_{evr} , and the threshold value θ_{thL} for the leftward steering may be a value $\theta_{evl} + \Delta\theta$ obtained by adding the predetermined value $\Delta\theta$ to the virtual rack end position θ_{evl} .

[0062] The control rotational displacement θ_r is, for example, set to zero (“0”) outside the impact mitigation control execution range (that is, $\theta_{thL} \leq \theta_{h1} \leq \theta_{thR}$), and, within the impact mitigation control execution range for the rightward steering, the larger a difference $\theta_{h1} - \theta_{thR}$ obtained by subtracting the threshold value θ_{thR} from the corrected steering angle θ_{h1} becomes, the larger the control rotational displacement θ_r is set. In contrast, within the impact mitigation control execution range for the leftward steering, the smaller a difference $\theta_{h1} - \theta_{thL}$ obtained by subtracting the threshold value θ_{thL} from the corrected steering angle θ_{h1} becomes (that is, the larger an absolute value $|\theta_{h1} - \theta_{thL}|$ becomes), the smaller the negative control rotational displacement θ_r may be set (that is, an absolute value $|\theta_r|$ becomes larger).

[0063] In other words, in a range where the corrected steering angle θ_{h1} is greater than the threshold value θ_{thR} , the positive control rotational displacement θ_r increases in accordance with increase in the corrected steering angle θ_{h1} , and, in a range where the corrected steering angle θ_{h1} is less than the threshold value θ_{thL} , the negative control rotational displacement θ_r decreases in accordance with decrease in the corrected steering angle θ_{h1} .

[0064] For example, the control rotational displacement setting unit 47 may set a difference $\theta_{h1} - \theta_{thR}$ as the control rotational displacement θ_r when the corrected steering angle θ_{h1} is greater than the threshold value θ_{thR} , and may set a difference $\theta_{h1} - \theta_{thL}$ as the control rotational displacement θ_r when the corrected steering angle θ_{h1} is less than the threshold value θ_{thL} .

[0065] FIG. 2 is now referred to. The differentiating unit 48 differentiates the steering angle θ_h detected by the steering angle sensor 14 and thereby calculates steering angular velocity ω .

[0066] The impact mitigation control unit 49 sets the impact mitigation control output I_{ref2} , based on the control rotational displacement θ_r and the steering angular velocity ω .

[0067] FIG. 4 is a block diagram illustrative of an example of a functional configuration of the impact mitigation control unit 49. The impact mitigation control unit 49 includes a spring constant table 60, multipliers 61 and 63, a viscosity constant table 62, an adder 64, an inverter 65, and a limiter 66.

[0068] The spring constant table 60 is a data table calculating a spring constant k_0 of a steering system. The spring constant k_0 has a characteristic of comparatively steeply increasing (nonlinearly increasing) at a middle portion of a change region as the control rotational displacement θ_r increases, as illustrated in FIG. 5A. Note that a characteristic in the case where the control rotational displacement θ_r is a negative value is a characteristic that is line symmetric to the characteristic in FIG. 5A with the spring-constant- k_0 axis (ordinate axis) as the symmetry axis.

[0069] The viscosity constant table 62 is a data table calculating a viscosity constant μ of the steering system. The viscosity constant μ has a characteristic of comparatively gently and gradually increasing (nonlinearly increasing) over the entire range as the control rotational displacement θ_r increases, as illustrated in FIG. 5B. Note that a characteristic in the case where the control rotational displacement θ_r is a negative value is a characteristic that is line symmetric to the characteristic in FIG. 5B with the viscosity-constant- μ axis (ordinate axis) as the symmetry axis.

[0070] The spring constant k_0 from the spring constant table 60 is multiplied by the control rotational displacement θ_r by the multiplier 61, and a multiplication result $k_0 \times \theta_r$ is input to the adder 64. In addition, the viscosity constant μ from the viscosity constant table 62 is multiplied by the steering angular velocity ω by the multiplier 63, and a multiplication result $\mu \times \omega$ is input to the adder 64. An addition result ($= k_0 \times \theta_r + \mu \times \omega$) by the adder 64 is input to the inverter 65 and the limiter 66, and the impact mitigation control output I_{ref2} the sign of which is inverted and the maximum value of which is limited is set.

[0071] Note that the configuration of the impact mitigation control unit 49 in FIG. 4 is only an exemplification and the present invention is not limited to the above-described configuration. The impact mitigation control unit 49 is only required to have a configuration capable of outputting the impact mitigation control output I_{ref2} that suppresses increase in the steering angle θ_h when the corrected steering angle θ_{h1} comes close to a rack end position.

[0072] Next, details of the terminal position learning unit 46 will be described. The terminal position learning unit 46 calculates, within a range of angle that the steering angle θ_h detected by the steering angle sensor 14 when rotational force applied to the turning mechanism is less than or equal to a first predetermined value can take, a steering angle that is positioned farthest from the steering angle neutral position (in the case of the positive steering angle θ_h , the maximum steering angle, and in the case of the negative steering angle θ_h , the minimum steering angle) as a first candidate θ_{m1} of the virtual rack end.

[0073] For example, the terminal position learning unit 46 may calculate a first candidate θ_{m1} of the virtual rack end when column output shaft torque T_c applied to the column output shaft 2o is less than or equal to a predetermined value $T1$.

[0074] The terminal position learning unit 46 may calculate, within a range of angle that the steering angle θ_h detected by the steering angle sensor 14 when the rotational

force applied to the turning mechanism is less than or equal to the first predetermined value and operational force acting on a steering operation unit is less than or equal to a third predetermined value can take, a steering angle that is positioned farthest from the steering angle neutral position as the first candidate θ_{m1} of the virtual rack end.

[0075] For example, the terminal position learning unit 46 may calculate the first candidate θ_{m1} when the column output shaft torque T_c is less than or equal to the predetermined value $T1$ and the steering torque T_h is less than or equal to a predetermined value $T2$.

[0076] In addition, the terminal position learning unit 46 calculates, within a range of an angle obtained by shifting the steering angle θ_h detected by the steering angle sensor 14 in a direction toward the steering angle neutral position by a second predetermined value, a steering angle that is positioned farthest from the steering angle neutral position (that is, in the case of the positive steering angle θ_h , an angle obtained by subtracting the second predetermined value from the maximum steering angle, and in the case of the negative steering angle θ_h , an angle obtained by adding the second predetermined value to the minimum steering angle) as a second candidate θ_{m2} of the virtual rack end. As the second predetermined value, for example, a maximum value that can be taken as error may be set.

[0077] The terminal position learning unit 46 selects a steering angle that is positioned farthest from the steering angle neutral position among the first candidate θ_{m1} and the second candidate θ_{m2} as the virtual rack end positions θ_{evr} and θ_{evl} .

[0078] This configuration can reduce influence of torsion or the like occurring due to torque and reduce error between the virtual rack end positions θ_{evr} and θ_{evl} and actual rack end positions.

[0079] FIG. 6 is a block diagram illustrative of an example of a functional configuration of the terminal position learning unit 46. The terminal position learning unit 46 includes an output shaft torque calculation unit 70, a selection unit 71, a first storage unit 72, delay units 73 and 77, a rate limiter 74, a corrected position calculation unit 75, a second storage unit 76, a third storage unit 78, a limiter 79, a stroke calculation unit 80, an offset error calculation unit 81, a subtracter 82, and a terminal position correction unit 83.

[0080] The output shaft torque calculation unit 70 calculates the column output shaft torque T_c applied to the column output shaft 20.

[0081] For example, the output shaft torque calculation unit 70 may calculate, as the column output shaft torque T_c , motor torque estimated by multiplying the current command value I_{ref3} for the motor 20 or the motor current value I_m detected by the current detector 50 by a motor torque constant and a reduction ratio of the reduction gear 3.

[0082] For example, the output shaft torque calculation unit 70 may calculate, as the column output shaft torque T_c , a sum of motor torque estimated by multiplying the current command value I_{ref3} for the motor 20 by the motor torque constant and the reduction ratio of the reduction gear 3 and the steering torque T_h detected by the torque sensor 10.

[0083] In addition, for example, the output shaft torque calculation unit 70 may also calculate, as the column output shaft torque T_c , a sum of motor torque estimated by multiplying the motor current value I_m detected by the current detector 50 by the motor torque constant and the reduction

ratio of the reduction gear 3 and the steering torque T_h detected by the torque sensor 10.

[0084] In addition, the output shaft torque calculation unit 70 may calculate motor angular acceleration by second-order differentiating a detected value of the angle sensor of the motor 20, estimate inertia torque by multiplying the motor angular acceleration by an inertia moment, and add the inertia torque to the column output shaft torque T_c calculated as described above.

[0085] The column output shaft torque T_c is an example of the “rotational force applied to the turning mechanism”. The steering torque T_h is an example of the “operational force acting on a steering operation unit of the vehicle”.

[0086] The subtracter 82 calculates the corrected steering angle θ_{h1} by subtracting the offset error Ofs calculated by the offset error calculation unit 81 from the steering angle θ_h detected by the steering angle sensor 14. The calculation of the offset error Ofs by the offset error calculation unit 81 will be described later.

[0087] The selection unit 71 selects one of the corrected steering angle θ_{h1} and output from the delay unit 73 depending on values of the column output shaft torque T_c and the steering torque T_h and outputs the selected one to the first storage unit 72. The delay unit 73 delays the first candidate θ_{m1} of the virtual rack end that is stored in and output from the first storage unit 72 and outputs the delayed first candidate θ_{m1} .

[0088] For example, the selection unit 71 may, when the column output shaft torque T_c is less than or equal to the predetermined value $T1$ and the steering torque T_h is less than or equal to the predetermined value $T2$, select the corrected steering angle θ_{h1} calculated from the detected steering angle θ_h and output the corrected steering angle θ_{h1} to the first storage unit 72 and, otherwise, output the output from the delay unit 73 to the first storage unit 72.

[0089] The first storage unit 72 stores the position farther from the steering angle neutral position than the other among the output from the delay unit 73 and the corrected steering angle θ_{h1} as the first candidate θ_{m1} of the virtual rack end.

[0090] Because of this configuration, when the corrected steering angle θ_{h1} that is calculated when the column output shaft torque T_c is less than or equal to the predetermined value $T1$ and the steering torque T_h is less than or equal to the predetermined value $T2$ is positioned farther from the steering angle neutral position than the first candidate θ_{m1} having been stored in the first storage unit 72 up to that time, the first candidate θ_{m1} stored in the first storage unit 72 is updated by the corrected steering angle θ_{h1} .

[0091] Note that the selection unit 71 may, when the column output shaft torque T_c is less than or equal to the predetermined value $T1$, select the corrected steering angle θ_{h1} and output the corrected steering angle θ_{h1} to the first storage unit 72 and, otherwise, output the output from the delay unit 73 to the first storage unit 72.

[0092] The rate limiter 74 accepts as input the first candidate θ_{m1} output from the first storage unit 72 and a steering angle θ_o output from the third storage unit. The rate limiter 74 limits a change rate of the first candidate θ_{m1} with respect to the steering angle θ_o delayed by a delay unit (not illustrated) and outputs a first candidate θ_{m1}' generated by limiting the change rate of the first candidate θ_{m1} to the third storage unit 78.

[0093] The corrected position calculation unit 75 calculates an angle obtained by shifting the corrected steering angle $\theta h1$ in the direction toward the steering angle neutral position by the second predetermined value. That is, when the corrected steering angle $\theta h1$ is positive, the corrected position calculation unit 75 outputs an angle obtained by subtracting the second predetermined value from the corrected steering angle $\theta h1$. When the corrected steering angle $\theta h1$ is negative, the corrected position calculation unit 75 outputs an angle obtained by adding the second predetermined value to the corrected steering angle $\theta h1$.

[0094] The second storage unit 76 stores the position farther from the steering angle neutral position than the other among the output from the corrected position calculation unit 75 and output from the delay unit 77 as the second candidate $\theta m2$ of the virtual rack end. The delay unit 77 delays the second candidate $\theta m2$ of the virtual rack end that is stored in and output from the second storage unit 76 and outputs the delayed second candidate $\theta m2$.

[0095] Because of this configuration, when the output from the corrected position calculation unit 75 (that is, an angle obtained by shifting the corrected steering angle $\theta h1$ in the direction toward the steering angle neutral position by the second predetermined value) is positioned farther from the steering angle neutral position than the second candidate $\theta m2$ having been stored in the second storage unit 76 up to that time, the second candidate $\theta m2$ stored in the second storage unit 76 is updated by the output from the corrected position calculation unit 75. The third storage unit 78 stores the position farther from the steering angle neutral position than the other among the first candidate $\theta m1'$ the change rate of which is limited by the rate limiter 74 and the second candidate $\theta m2$ as the steering angle θo and outputs the steering angle θo .

[0096] The limiter 79 limits the magnitude of the steering angle θo , which is output from the third storage unit 78, and outputs the limited steering angle θo as the virtual rack end positions θ_{evr} and θ_{evl} .

[0097] With reference to FIGS. 7A and 7B, a learning example of a virtual rack end according to the present embodiment will be described. For simplification of description, it is assumed that the offset error Ofs is 0 (that is, (steering angle θh)=(corrected steering angle $\theta h1$) holds), and a case is described where when the steering angle θh that is detected when the column output shaft torque T_c is less than or equal to the predetermined value $T1$ is positioned farther from the steering angle neutral position than the first candidate $\theta m1$ having been stored in the first storage unit 72 up to that time, the first candidate $\theta m1$ is updated.

[0098] FIG. 7A is an explanatory diagram of an example of change in the column output shaft torque T_c associated with change in the steering angle θh . Arrows in the drawing indicate steering directions.

[0099] At the time of additional turning of the steering wheel, the column output shaft torque T_c exceeds the predetermined value $T1$ when the steering angle θh exceeds $\theta1$, and at the time of subsequent reverse turning of the steering wheel, the column output shaft torque T_c becomes less than the predetermined value $T1$ when the steering angle θh becomes less than $\theta2$ ($\theta2 > \theta1$).

[0100] FIG. 7B is an explanatory diagram of an example of a learned value of a virtual rack end in the rightward steering when the column output shaft torque T_c in FIG. 7A is generated. The dashed line, the dashed-dotted line, the

dashed-two-dotted line, and the solid line indicate the steering angle θh , the first candidate $\theta m1$, the second candidate $\theta m2$, and the output θo from the third storage unit 78 (the virtual rack end positions θ_{evr} and θ_{evl} before being limited by the limiter 79), respectively. Note that the dashed-dotted line and the dashed-two-dotted line are illustrated at slightly shifted positions lest the dashed-dotted line and the dashed-two-dotted line overlap other lines.

[0101] When, at time $t1$, the steering angle θh increases and the additional turning of the steering wheel is started, while the column output shaft torque T_c is less than or equal to the predetermined value $T1$ (that is, while the steering angle θh is less than or equal to $\theta1$), the steering angle θh (dashed line) is learned as the first candidate $\theta m1$ (dashed-dotted line). During the additional turning of the steering wheel, the first candidate $\theta m1$ (dashed-dotted line) increases to $\theta1$.

[0102] In addition, an angle obtained by subtracting the second predetermined value from the steering angle θh is learned as the second candidate $\theta m2$ (dashed-two-dotted line).

[0103] Thus, while the first candidate $\theta m1'$ the change rate of which is limited by the rate limiter 74 is greater than the second candidate $\theta m2$ (dashed-two-dotted line) (from time $t1$ to time $t2$), the first candidate $\theta m1'$ is selected as the output θo (solid line) from the third storage unit 78, and when, at time $t2$, the second candidate $\theta m2$ exceeds the first candidate $\theta m1'$, the second candidate $\theta m2$ is selected as the output θo (solid line).

[0104] Subsequently, when, at time $t3$, the steering angle θh ceases to increase and stays at a constant value, the second candidate $\theta m2$ (dashed-two-dotted line) also ceases to increase. Thus, thereafter, the first candidate $\theta m1'$ the change rate of which is limited by the rate limiter 74 is selected as the output θo (solid line) from the third storage unit 78.

[0105] Since, as described above, the first candidate $\theta m1$ (dashed-dotted line) increases to $\theta1$, the output θo (solid line) from the third storage unit 78 also increases to $\theta1$ behind the first candidate $\theta m1$. When, at time $t4$, the output θo (solid line) reaches $\theta1$, the output θo ceases to increase.

[0106] When, subsequently, the steering angle θh starts to decrease and the reverse turning of the steering wheel is started, the steering angle θh decreases to $\theta2$ at time $t5$. Then, the column output shaft torque T_c becomes less than or equal to the predetermined value $T1$. Thus, the angle $\theta2$ is learned as the first candidate $\theta m1$ (dashed-dotted line).

[0107] Thus, the first candidate $\theta m1'$ the change rate of which is limited by the rate limiter 74 starts to increase and is selected as the output θo (solid line) from the third storage unit 78. The output θo (solid line) increases until reaching $\theta2$ at time $t6$ and subsequently becomes constant.

[0108] When comparing the output θo (solid line) from the third storage unit 78 that is learned as described above, that is, the virtual rack end positions θ_{evr} and θ_{evl} before being limited by the limiter 79, with a case where an angle obtained by simply subtracting the second predetermined value (for example, a maximum error estimation value) from the steering angle θh is learned (dashed-two-dotted line), the output θo from the third storage unit 78 can be learned as a steering angle positioned farther from a steering angle neutral point than the other. Thus, a steering angle positioned closer to an actual rack end position can be learned as the virtual rack end positions θ_{evr} and θ_{evl} .

[0109] Next, the learning state determination unit 51 will be described. FIG. 2 is now referred to. The learning state determination unit 51 determines, based on the virtual rack end positions θ_{evr} and θ_{evl} output from the terminal position learning unit 46, a state of learning of a virtual rack end position by the terminal position learning unit 46.

[0110] The learning state determination unit 51 outputs, depending on a determination result on the state of learning of a virtual rack end position, one of 0 and positive limiting values Limit1 and Limit2 to the impact mitigation control output limiting unit 52 as a limiting value to limit the upper limit of the impact mitigation control output Iref2 at the time of leftward steering.

[0111] The limiting value Limit2 is a larger value than the limiting value Limit1, and may, for example, be set to a sufficiently large value to effectively prevent impact and hit sound (abnormal noise) due to end-abutting. On the other hand, the limiting value Limit1 may be set to a value that, although allowing a certain level of impact and hit sound (abnormal noise), can prevent damage to the steering mechanism due to end-abutting.

[0112] The learning state determination unit 51 also outputs, depending on a determination result on the state of learning of a virtual rack end position, one of 0 and negative limiting values -Limit1 and -Limit2 to the impact mitigation control output limiting unit 52 as a limiting value to limit the lower limit of the impact mitigation control output Iref2 at the time of rightward steering.

[0113] Specifically, as initial values before start of learning of the virtual rack end positions θ_{evr} and θ_{evl} , a positive initial value θ_{int} and a negative initial value $-\theta_{int}$ are stored, respectively, in the first storage unit 72, the second storage unit 76, and the third storage unit 78.

[0114] The initial values θ_{int} and $-\theta_{int}$ may be appropriately set in such a way that there is no possibility that the initial values θ_{int} and $-\theta_{int}$ are positioned on the outer side of actual rack end positions (that is, in such a way that there is no possibility that the initial values θ_{int} and $-\theta_{int}$ are positioned farther from the steering angle neutral point than the actual rack end positions). For example, the initial values θ_{int} and $-\theta_{int}$ may be set in such a manner that $\theta_{int} = (\text{rack stroke minimum value } St_{min}) - (\text{rack end maximum value } fev_{max})$ holds.

[0115] In the setting, the rack stroke minimum value St_{min} may be set to a minimum value of variation in values that can be calculated as a rack stroke between the virtual rack end positions θ_{evr} and θ_{evl} (for example, a lower limit of manufacturing tolerance).

[0116] In addition, the “rack end maximum value θ_{evmax} ” is a maximum value of absolute values of values that can be learned as the virtual rack end positions θ_{evr} and θ_{evl} and may be set in such a manner that the rack end maximum value $\theta_{evmax} = (\text{rack stroke maximum value } St_{max})/2 + (\text{estimated value of offset error between rack neutral position and steering angle neutral position})$ holds.

[0117] In addition, the rack stroke maximum value St_{max} is a maximum value of variation in values that can be calculated as the rack stroke between the virtual rack end positions θ_{evr} and θ_{evl} and may, for example, be set to a value obtained by adding learning error of the virtual rack end positions θ_{evr} and θ_{evl} to an upper limit of manufacturing tolerance.

[0118] When the right virtual rack end position θ_{evr} output from the terminal position learning unit 46 is less than

a predetermined learning threshold value θ_{lth} , the learning state determination unit 51 determines that learning of the right virtual rack end position θ_{evr} has not been performed and outputs “0” as a limiting value to limit the lower limit of the impact mitigation control output Iref2.

[0119] As the “learning threshold value θ_{lth} ”, a minimum value of the absolute values of values that can be learned as the virtual rack end positions θ_{evr} and θ_{evl} may be set, and the “learning threshold value θ_{lth} ” may, for example, be set in such a manner that the learning threshold value $\theta_{lth} = (\text{rack stroke minimum value } St_{min})/2 - (\text{estimated value of offset error between rack neutral position and steering angle neutral position})$ holds.

[0120] Likewise, when the left virtual rack end position θ_{evl} is greater than a negative learning threshold value $-\theta_{lth}$ (that is, the absolute value $|\theta_{evl}|$ is less than the absolute value $|\theta_{lth}|$), the learning state determination unit 51 determines that learning of the left virtual rack end position θ_{evl} has not been performed and outputs “0” as a limiting value to limit the upper limit of the impact mitigation control output Iref2.

[0121] When the right virtual rack end position θ_{evr} is greater than or equal to the predetermined learning threshold value θ_{lth} , the learning state determination unit 51 determines that the learning of the right virtual rack end position θ_{evr} has been performed and outputs “-Limit1” as a limiting value to limit the lower limit of the impact mitigation control output Iref2.

[0122] Likewise, when the left virtual rack end position θ_{evl} is less than or equal to the negative learning threshold value $-\theta_{lth}$ (that is, the absolute value $|\theta_{evl}|$ is greater than or equal to the absolute value $|\theta_{lth}|$), the learning state determination unit 51 determines that the learning of the left virtual rack end position θ_{evl} has been performed and outputs “Limit1” as a limiting value to limit the upper limit of the impact mitigation control output Iref2.

[0123] Further, the learning state determination unit 51 calculates distance between the right virtual rack end position θ_{evr} and the left virtual rack end position θ_{evl} as rack stroke St .

[0124] When the absolute values of learned values of the virtual rack end positions θ_{evr} and θ_{evl} become large and the rack stroke St becomes longer than the rack stroke minimum value St_{min} , the learning state determination unit 51 determines that the learning of the virtual rack end positions θ_{evr} and θ_{evl} has been completed and outputs “-Limit2” and “Limit2” as a limiting value to limit the lower limit of the impact mitigation control output Iref2 and a limiting value to limit the upper limit of the impact mitigation control output Iref2, respectively.

[0125] Next, an example of a manner in which, depending on a change in the state of learning of the virtual rack end positions, the limiting value that the learning state determination unit 51 outputs changes will be described.

[0126] FIGS. 8A, 8B, 8C, 8D, 8E, and 8F are a conceptual diagram of actual rack end positions, a conceptual diagram of a state before start of learning of virtual rack end positions, a conceptual diagram of a state in which the right virtual rack end position θ_{evr} has been learned, a conceptual diagram of a state in which the left virtual rack end position θ_{evl} has been learned, a conceptual diagram of a state in which the learning of the virtual rack ends θ_{evr} and θ_{evl} is considered to have been completed, and a conceptual dia-

gram of a state in which the learning of the virtual rack ends has been continued to the vicinities of the actual rack ends, respectively.

[0127] In addition, limiting values in a table in FIG. 10 indicate limiting values that the learning state determination unit 51 outputs in the states illustrated in FIGS. 8B to 8F.

[0128] In FIGS. 8B to 8F, “0 [deg]” indicates the steering angle neutral position. The same applies to FIGS. 9B to 9H, which are described later.

[0129] In FIGS. 8B to 8F, the steering angle neutral position substantially coincides with the rack neutral position (the center position between the actual rack end positions).

[0130] In the state before start of learning of the virtual rack end positions (FIG. 8B), the right virtual rack end position θ_{vr} , which is output from the terminal position learning unit 46, is θ_{int} with reference to the steering angle neutral position and less than the learning threshold value θ_{lth} .

[0131] Therefore, the learning state determination unit 51 determines that the learning of the right virtual rack end position θ_{vr} has not been performed and outputs “0” as a limiting value to limit the lower limit of the impact mitigation control output Iref2 (see FIG. 10).

[0132] In addition, the left virtual rack end position θ_{vl} , which is output from the terminal position learning unit 46, is $-\theta_{int}$ with reference to the steering angle neutral position and greater than the learning threshold value $-\theta_{lth}$. Therefore, the learning state determination unit 51 determines that the learning of the left virtual rack end position θ_{vl} has not been performed and outputs “0” as a limiting value to limit the upper limit of the impact mitigation control output Iref2 (see FIG. 10).

[0133] Subsequently, as illustrated in FIG. 8C, the right virtual rack end position θ_{vr} is learned. Since the right virtual rack end position θ_{vr} is greater than or equal to the learning threshold value θ_{lth} , the learning state determination unit 51 determines that the learning of the right virtual rack end position θ_{vr} has been performed and outputs “-Limit1” as a limiting value to limit the lower limit of the impact mitigation control output Iref2. On the other hand, since the left virtual rack end position θ_{vl} has not changed, the learning state determination unit 51 outputs “0” as a limiting value to limit the upper limit of the impact mitigation control output Iref2 (see FIG. 10).

[0134] Subsequently, as illustrated in FIG. 8D, the left virtual rack end position θ_{vl} is learned. Since the left virtual rack end position θ_{vl} becomes less than or equal to the negative learning threshold value $-\theta_{lth}$, the learning state determination unit 51 determines that the learning of the left virtual rack end position θ_{vl} has been performed. However, since the rack stroke St is less than or equal to the rack stroke minimum value St_{min} , the learning state determination unit 51 does not determine that the learning of the virtual rack end positions θ_{vr} and θ_{vl} has been completed.

[0135] Therefore, the learning state determination unit 51 outputs “Limit1” as a limiting value to limit the upper limit of the impact mitigation control output Iref2. The learning state determination unit 51 also outputs “-Limit1” as a limiting value to limit the lower limit of the impact mitigation control output Iref2.

[0136] Subsequently, as illustrated in FIG. 8E, further learning of the right virtual rack end position θ_{vr} causes the rack stroke St to become longer than the rack stroke mini-

mum value St_{min} . Therefore, the learning state determination unit 51 determines that the learning of the virtual rack end positions θ_{vr} and θ_{vl} has been completed and outputs “Limit2” and “-Limit2” as limiting values to limit the upper limit and the lower limit of the impact mitigation control output Iref2, respectively.

[0137] Subsequently, repeating the learning of the virtual rack end positions θ_{vr} and θ_{vl} causes the virtual rack end positions θ_{vr} and θ_{vl} to come close to the actual rack end positions (see FIG. 8F).

[0138] The learning state determination unit 51 outputs “Limit2” and “-Limit2” as limiting values to limit the upper limit and the lower limit of the impact mitigation control output Iref2, respectively, until the rack stroke St exceeds the rack stroke maximum value St_{max} .

[0139] Next, operation in the case where offset error occurs between the rack neutral position and the steering angle neutral position will be described. FIG. 9A is a conceptual diagram of the actual rack end positions, FIG. 9B is the same drawing as FIG. 8F and illustrates a state in which there is no offset error, and FIG. 9C is a conceptual diagram of a state immediately after offset error has occurred.

[0140] The steering angle neutral position in FIG. 9B substantially coincides with the rack neutral position (the center position between the actual rack end positions), and the steering angle neutral position in FIG. 9C is shifted to the right side from the rack neutral position by $\Delta\theta$. That is, when the steering wheel is steered to the left side of the steering angle neutral position by $\Delta\theta$, the rack 5b is positioned at the rack neutral position.

[0141] When offset error occurs, the impact mitigation control cannot be normally performed. In the example in FIG. 9C, since the right virtual rack end position θ_{vr} is positioned on the outer side of the actual rack end position, necessary reduction in impact and abnormal noise cannot be performed.

[0142] Thus, the terminal position learning unit 46, as described above, estimates an offset error Ofs between the rack neutral position and the steering angle neutral position and outputs the corrected steering angle θ_{h1} , to which the steering angle θ_h detected by the steering angle sensor 14 is corrected by subtracting the offset error Ofs from the steering angle θ_h .

[0143] FIG. 6 is now referred to. The stroke calculation unit 80 calculates the rack stroke St . The offset error calculation unit 81 compares the rack stroke St with the rack stroke maximum value St_{max} .

[0144] When the rack stroke St exceeds the rack stroke maximum value St_{max} , the offset error calculation unit 81 determines that offset error has occurred. The offset error calculation unit 81 calculates a difference obtained by subtracting the rack stroke maximum value St_{max} from the rack stroke St as the offset error $Ofs = (St - St_{max})$.

[0145] FIG. 9D is a conceptual diagram of a state in which, after offset error has occurred, the left virtual rack end position θ_{vl} is learned.

[0146] Since the steering angle neutral position is shifted to the right side of the rack neutral position, when the left virtual rack end position θ_{vl} is newly learned, the rack stroke St between the virtual rack end positions θ_{vr} and θ_{vl} exceeds the rack stroke maximum value St_{max} . The offset error calculation unit 81 calculates a difference

obtained by subtracting the rack stroke maximum value St_{max} from the rack stroke St (rack stroke St -rack stroke maximum value St_{max}) as the offset error Ofs .

[0147] In the following description, a virtual rack end position that is one of the right and left virtual rack end positions that is learned when a rack stroke St exceeding the rack stroke maximum value St_{max} is calculated is sometimes referred to as “one virtual rack end position”. In addition, a virtual rack end position that is one of the right and left virtual rack end positions that is not the one virtual rack end position is sometimes referred to as “the other virtual rack end position”.

[0148] When, as in the example in FIG. 9C, the steering angle neutral position is shifted to the right side from the rack neutral position, the left virtual rack end position serves as one virtual rack end position and the right virtual rack end position serves as the other virtual rack end position. Conversely, when the steering angle neutral position is shifted to the left side from the rack neutral position, the right virtual rack end position serves as one virtual rack end position and the left virtual rack end position serves as the other virtual rack end position.

[0149] FIG. 6 is now referred to. The subtracter 82 calculates the corrected steering angle θ/h by subtracting the offset error Ofs calculated by the offset error calculation unit 81 from the steering angle θ_h detected by the steering angle sensor 14.

[0150] The terminal position correction unit 83 corrects the first candidate $\theta m1$ that the first storage unit 72 stores, the second candidate $\theta m2$ that the second storage unit 76 stores, and the steering angle θ that the third storage unit 78 stores according to the offset error Ofs . With reference to FIG. 9E, the correction processing of the candidates and the steering angle will be described.

[0151] The subtracter 82 subtracting the offset error Ofs from the steering angle θ_h causes the steering angle neutral position (position at “0 [deg]”) to move, as illustrated in FIG. 9E.

[0152] On the other hand, since the left virtual rack end position θ_{vl} (that is, one virtual rack end position) that was learned in FIG. 9D is a learned value before the steering angle neutral position, which serves as the base point, moves, when the steering angle neutral position is moved as illustrated in FIG. 9E, it is necessary to correct the left virtual rack end position θ_{vl} in association with this movement.

[0153] The terminal position correction unit 83 corrects the first candidate $\theta m1$ of the left virtual rack end position stored in the first storage unit 72 with the offset error Ofs . Since the left virtual rack end position is a negative value, the terminal position correction unit 83 corrects the left virtual rack end position by adding the offset error Ofs . The terminal position correction unit 83 also likewise corrects the second candidate $\theta m2$ and the steering angle θ that are stored in the second storage unit 76 and the third storage unit 78, respectively.

[0154] When the one virtual rack end position is the right virtual rack end position (that is, when the virtual rack end position is a positive value), the terminal position correction unit 83 corrects the right virtual rack end position by subtracting the offset error Ofs .

[0155] In addition, the terminal position correction unit 83 corrects (resets) the other virtual rack end position (in the example in FIG. 9E, the right virtual rack end position θ_{vr})

in such a way that the rack stroke St between the virtual rack end positions θ_{vr} and θ_{vl} coincides with the predetermined rack stroke minimum value St_{min} . This configuration can correct the other virtual rack end position to a position on the inner side of the actual rack end position.

[0156] FIG. 9F is now referred to. When a new left virtual rack end position θ_{vl} is further learned, the offset error calculation unit 81 calculates a change amount $\Delta\theta_{vl}$ of the virtual rack end position θ_{vl} between before and after update.

[0157] The offset error calculation unit 81, by adding the change amount $\Delta\theta_{vl}$ to the offset error Ofs before the new left virtual rack end position θ_{vl} is learned, updates the offset error Ofs . This update causes the steering angle neutral position to further move by the change amount $\Delta\theta_{vl}$.

[0158] The terminal position correction unit 83 corrects the first candidate $\theta m1$ of the one virtual rack end position (the left virtual rack end position) that is stored in the first storage unit 72 with the change amount $\Delta\theta_{vl}$. Since the left virtual rack end position is a negative value, the terminal position correction unit 83 corrects the left virtual rack end position by adding the change amount $\Delta\theta_{vl}$. The terminal position correction unit 83 also likewise corrects the second candidate $\theta m2$ and the steering angle θ that are stored in the second storage unit 76 and the third storage unit 78, respectively.

[0159] When the one virtual rack end position is the right virtual rack end position (that is, the virtual rack end position is a positive value), the terminal position correction unit 83 corrects the first candidate $\theta m1$ by subtracting the change amount $\Delta\theta_{vl}$.

[0160] In addition, the terminal position correction unit 83 corrects (resets) the other virtual rack end position (in the example in FIG. 9F, the right virtual rack end position θ_{vr}) in such a way that the rack stroke St between the virtual rack end positions θ_{vr} and θ_{vl} coincides with the rack stroke minimum value St_{min} .

[0161] FIG. 9G is now referred to. When a new right virtual rack end position θ_{vr} (that is, the other virtual rack end position) is learned, the offset error calculation unit 81 does not update the offset error Ofs . That is, the steering angle neutral position is not moved.

[0162] In addition, the terminal position correction unit 83 also does not correct the first candidate $\theta m1$ that the first storage unit 72 stores, the second candidate $\theta m2$ that the second storage unit 76 stores, and the steering angle θ that the third storage unit 78 stores. Because of this configuration, only the right virtual rack end position θ_{vr} is updated in such a manner as to move away from the steering angle neutral position.

[0163] FIG. 9H is now referred to. After a new right virtual rack end position θ_{vr} has been learned in FIG. 9G, even when a new left virtual rack end position θ_{vl} (that is, the one virtual rack end position) is further learned, the terminal position correction unit 83 does not correct (reset) the right virtual rack end position θ_{vr} in such a way that the rack stroke St coincides with the predetermined rack stroke minimum value St_{min} .

[0164] As with FIG. 9F, the offset error calculation unit 81 updates the offset error Ofs (that is, the steering angle neutral position is modified), which causes the steering angle neutral position to move, and the terminal position correction unit 83 corrects the left virtual rack end position θ_{vl} . On

this occasion, the right virtual rack end position θ_{vr} is corrected by adding the offset error Ofs to the virtual rack end position θ_{vr} .

[0165] In addition, when a new right virtual rack end position θ_{vr} (that is, the other virtual rack end position) is further learned, the steering angle neutral position and the left virtual rack end position θ_{vl} (that is, the one virtual rack end position) are not changed and only the right virtual rack end position θ_{vr} changes.

[0166] Next, operation of the learning state determination unit 51 in the case where offset error occurs will be described. The learning state determination unit 51 determines that offset error has occurred when the rack stroke St , which is calculated from the right virtual rack end position θ_{vr} and the left virtual rack end position θ_{vl} , is longer than the rack stroke maximum value St_{max} .

[0167] When determining that offset error has occurred, the learning state determination unit 51 resets each of the limiting values to limit the upper limit and the lower limit of the impact mitigation control output $Iref2$ to “0”

[0168] Subsequently, when the right virtual rack end position θ_{vr} is, as with the above description, greater than or equal to the predetermined learning threshold value θ_{lth} , the learning state determination unit 51 outputs “-Limit1” as a limiting value to limit the lower limit of the impact mitigation control output $Iref2$. When the left virtual rack end position θ_{vl} is less than or equal to the negative learning threshold value $-\theta_{lth}$, the learning state determination unit 51 outputs “Limit1” as a limiting value to limit the upper limit of the impact mitigation control output $Iref2$. When the rack stroke St becomes longer than the rack stroke minimum value St_{min} , the learning state determination unit 51 outputs “Limit2” and “-Limit2” as limiting values to limit the upper limit and the lower limit of the impact mitigation control output $Iref2$, respectively.

[0169] With reference to FIGS. 9A to 9F and 10, an example of limiting values that the learning state determination unit 51 outputs when offset error has occurred will be described. Limiting values in the table in FIG. 10 indicate limiting values that the learning state determination unit 51 outputs in the states illustrated in FIGS. 9B to 9H.

[0170] FIG. 9B is the same as FIG. 8F and illustrates a state before offset error occurs. The learning state determination unit 51 outputs “Limit2” and “-Limit2” as limiting values to limit the upper limit and the lower limit of the impact mitigation control output $Iref2$, respectively.

[0171] Subsequently, when offset error occurs due to a reason such as incorrect assembly of the intermediate shaft 4, the state in FIG. 9B transitions to the state illustrated in FIG. 9C. Since, in this stage, learning of new virtual rack ends θ_{vr} and θ_{vl} has not been performed, the value of the rack stroke St that the learning state determination unit 51 calculates stays at the same value as that in the state in FIG. 9B. Therefore, the learning state determination unit 51 has not determined that offset error occurred and has not reset the limiting values to “0”. Thus, the learning state determination unit 51 outputs “Limit2” and “-Limit2”.

[0172] When, in FIG. 9D, a new left virtual rack end position θ_{vl} is learned, the rack stroke St becomes longer than the rack stroke maximum value St_{max} . Thus, the learning state determination unit 51 resets the limiting values to limit the upper limit and the lower limit of the impact mitigation control output $Iref2$ to “0”

[0173] In addition, since the left virtual rack end position θ_{vl} is less than or equal to the negative learning threshold value $-\theta_{lth}$ as in FIG. 9E, the learning state determination unit 51 determines that the learning of the left virtual rack end position θ_{vl} has been performed and outputs “Limit1” as a limiting value to limit the upper limit of the impact mitigation control output $Iref2$.

[0174] On the other hand, since the right virtual rack end position θ_{vr} is corrected (reset) in such a way that the rack stroke St between the virtual rack end positions θ_{vr} and θ_{vl} coincides with the rack stroke minimum value St_{min} , the rack stroke St does not become longer than the rack stroke minimum value St_{min} . Therefore, it is not determined that the learning of the virtual rack end positions θ_{vr} and θ_{vl} has been completed, and since the right virtual rack end position θ_{vr} is less than the learning threshold value θ_{lth} , output of “0” as a limiting value to limit the lower limit of the impact mitigation control output $Iref2$ is maintained. The same applies to the state in FIG. 9F.

[0175] FIG. 9G is now referred to. When a new right virtual rack end position θ_{vr} (that is, the other virtual rack end position) is learned and the rack stroke St becomes longer than the rack stroke minimum value St_{min} , the learning state determination unit 51 determines that the learning of the virtual rack end positions θ_{vr} and θ_{vl} has been completed and outputs “Limit2” and “-Limit2” as limiting values to limit the upper limit and the lower limit of the impact mitigation control output $Iref2$, respectively.

[0176] In FIG. 9H, the learning state determination unit 51 likewise outputs “Limit2” and “-Limit2” as limiting values to limit the upper limit and the lower limit of the impact mitigation control output $Iref2$, respectively.

[0177] As described above, the turning control device of the present embodiment is capable of limiting the impact mitigation control output $Iref2$ in a stepwise manner depending on a degree of learning of the virtual rack end positions, based on a comparison result between a learned virtual rack end position and the learning threshold value θ_{lth} and a comparison result between the rack stroke St calculated from the learned virtual rack end position and the rack stroke minimum value St_{min} . Because of this configuration, it is possible to learn virtual rack end positions while preventing damage to the steering mechanism due to end-abutting.

[0178] Next, the relearning determination unit 53 will be described. FIG. 2 is now referred to. The relearning determination unit 53 determines necessity of relearning of the virtual rack end position θ_{vr} or θ_{vl} depending on whether or not occurrence of end-abutting is detected. In the following description, the virtual rack end positions θ_{vr} and θ_{vl} are sometimes collectively referred to as “virtual rack end positions θ_{ev} ”.

[0179] When the relearning of the virtual rack end positions θ_{ev} is to be performed, the relearning determination unit 53 outputs a relearning command signal Cmd commanding the terminal position learning unit 46 to perform relearning. Because of this configuration, the virtual rack end positions θ_{ev} can be relearned when, for example, in association with a rack shaft being replaced after the virtual rack end positions θ_{ev} are learned, end-abutting occurs due to a reason, such as a rack shaft that has a length different from length of a rack shaft that should be originally mounted being mistakenly mounted or a mounting position of a rack shaft having changed between before and after replacement of the rack shaft.

[0180] FIG. 11 is a block diagram illustrative of an example of a functional configuration of the relearning determination unit 53. The relearning determination unit 53 includes an end-abutting detection unit 90, an end-abutting steering angle range determination unit 91, a counting unit 92, and a relearning unit 93.

[0181] The end-abutting detection unit 90 detects occurrence of end-abutting, based on the column output shaft torque T_c and the steering torque T_h and retains an end-abutting steering angle θ_{abt} that is a steering angle when occurrence of end-abutting is detected.

[0182] For example, the end-abutting detection unit 90 retains a steering angle θ_{dt} when the steering torque T_h is greater than or equal to a predetermined threshold value T_{th1} (for example, 8 Nm) and, based on the retained steering angle θ_{dt} , determines necessity of detecting the end-abutting steering angle θ_{abt} . For example, the end-abutting detection unit 90 may determine that detection of the end-abutting steering angle θ_{abt} is necessary when an absolute value $|\theta_{ev}-\theta_{dt}|$ of a difference between the steering angle θ_{dt} and one of the virtual rack end positions θ_{ev} is greater than a steering angle threshold value θ_{tha1} and determine that the detection of the end-abutting steering angle θ_{abt} is not necessary when the absolute value $|\theta_{ev}-\theta_{dt}|$ of the difference between the steering angle θ_{dt} and the virtual rack end position θ_{ev} is less than or equal to the steering angle threshold value θ_{tha1} .

[0183] FIG. 12A is now referred to. FIG. 12A illustrates as an example a case where the steering wheel 1 is steered to the right. In processing when the steering wheel 1 is steered to the left, the right virtual rack end position θ_{evr} in the following description is replaced by the left virtual rack end position θ_{evl} .

[0184] When the steering angle when the steering torque T_h is greater than or equal to the predetermined threshold value T_{th1} is a steering angle θ_{dt1} illustrated in the drawing, the end-abutting detection unit 90 determines that the detection of the end-abutting steering angle θ_{abt} is not necessary since the absolute value $|\theta_{evr}-\theta_{dt1}|$ of the difference between the virtual rack end position θ_{evr} and the steering angle θ_{dt1} is less than or equal to the steering angle threshold value θ_{tha1} . On the other hand, when the steering angle when the steering torque T_h is greater than or equal to the predetermined threshold value T_{th1} is a steering angle θ_{dt2} illustrated in the drawing, the end-abutting detection unit 90 determines that the detection of the end-abutting steering angle θ_{abt} is necessary since the absolute value $|\theta_{evr}-\theta_{dt2}|$ of the difference between the virtual rack end position θ_{evr} and the steering angle θ_{dt2} is greater than the steering angle threshold value θ_{tha1} .

[0185] When determining that the detection of the end-abutting steering angle θ_{abt} is necessary, the end-abutting detection unit 90 retains a steering angle θ_{cd} when, after the steering torque T_h has become greater than or equal to the predetermined threshold value T_{th1} , the column output shaft torque T_c becomes less than a predetermined threshold value T_{th2} (for example, 35 Nm) and the steering torque T_h becomes less than the predetermined threshold value T_{th1} , as a candidate of the end-abutting steering angle θ_{abt} (hereinafter, sometimes referred to as “end-abutting steering angle candidate”).

[0186] The end-abutting detection unit 90 acquires an end-abutting steering angle candidate θ_{cd} as the end-abutting steering angle θ_{abt} when an absolute value $|\theta_{ev}-\theta_{cd}|$ of

a difference between the end-abutting steering angle candidate θ_{cd} and one of the virtual rack end positions θ_{ev} is greater than a steering angle threshold value θ_{tha2} . When the absolute value $|\theta_{ev}-\theta_{cd}|$ is less than or equal to the steering angle threshold value θ_{tha2} , the end-abutting detection unit 90 discards the end-abutting steering angle candidate θ_{cd} without acquiring the end-abutting steering angle candidate θ_{cd} as the end-abutting steering angle θ_{abt} . For example, the steering angle threshold value θ_{tha2} may be set to a value a predetermined margin Δ smaller than the steering angle threshold value θ_{tha1} . For example, the predetermined margin Δ may be set according to detection error of the steering angle θ_h .

[0187] FIG. 12B is now referred to. FIG. 12B illustrates as an example the case where the steering wheel 1 is steered to the right. In the processing when the steering wheel 1 is steered to the left, the right virtual rack end position θ_{evr} in the following description is replaced by the left virtual rack end position θ_{evl} .

[0188] In a case of an end-abutting steering angle candidate θ_{cd1} , the end-abutting detection unit 90 discards the end-abutting steering angle candidate θ_{cd1} without acquiring the end-abutting steering angle candidate θ_{cd1} as the end-abutting steering angle θ_{abt} since an absolute value $|\theta_{evr}-\theta_{cd1}|$ of a difference between the virtual rack end position θ_{evr} and the end-abutting steering angle candidate θ_{cd1} is less than or equal to the steering angle threshold value θ_{tha2} . On the other hand, in a case of an end-abutting steering angle candidate θ_{cd2} , the end-abutting detection unit 90 acquires the end-abutting steering angle candidate θ_{cd2} as the end-abutting steering angle θ_{abt} since an absolute value $|\theta_{evr}-\theta_{cd2}|$ of a difference between the virtual rack end position θ_{evr} and the end-abutting steering angle candidate θ_{cd2} is greater than the steering angle threshold value θ_{tha2} .

[0189] Next, the end-abutting detection unit 90 calculates a rack stroke approximate value Sta , based on the acquired end-abutting steering angle θ_{abt} . When the rack stroke approximate value Sta is not a value within an allowable range, the end-abutting detection unit 90 may exclude the acquired end-abutting steering angle θ_{abt} from targets of determination processing of variation in end-abutting steering angles θ_{abt} performed by the end-abutting steering angle range determination unit 91, which will be described later, and discard the acquired end-abutting steering angle θ_{abt} .

[0190] FIG. 13A illustrates a calculation example of the rack stroke approximate value Sta when in a case where the steering wheel 1 is steered to one of the left side and the right side, end-abutting does not occur and in a case where the steering wheel 1 is steered to the other of the left side and the right side, end-abutting occurs and the end-abutting steering angle θ_{abt} is acquired. FIG. 13B illustrates a calculation example of the rack stroke approximate value Sta when both in the case where the steering wheel 1 is steered to the left side and in the case where the steering wheel 1 is steered to the right side, end-abutting occurs and an end-abutting steering angle θ_{abtl} and an end-abutting steering angle θ_{abtr} are acquired both on the left side and on the right side, respectively.

[0191] When, as illustrated in FIG. 13A, the end-abutting steering angle θ_{abt} is acquired when the steering wheel 1 is steered to the right side, the end-abutting detection unit 90 calculates an absolute value of a difference between the left

virtual rack end position θ_{evl} and the end-abutting steering angle θ_{abt} on the right side as the rack stroke approximate value Sta .

[0192] When error between the rack stroke approximate value Sta and a predetermined value does not fall within a threshold value, it is considered that the vehicle is in a state in which one of the steered wheels **8L** and **8R** collides with a curb or the like when the steering wheel **1** is steered and the steering wheel **1** cannot be steered to a rack end. Thus, when error between the rack stroke approximate value Sta and the predetermined value does not fall within the threshold value, the end-abutting detection unit **90** may discard the steering angle θ_{abt} without using the steering angle θ_{abt} for determination of necessity of relearning.

[0193] Note that in the processing in a case where the end-abutting steering angle θ_{abt} is acquired when the steering wheel **1** is steered to the left, the left virtual rack end position θ_{evl} in the above description is replaced by the right virtual rack end position θ_{evr} .

[0194] In the following description, an end-abutting steering angle θ_{abt} acquired when the steering wheel **1** is steered to the left side is sometimes referred to as “left end-abutting steering angle”, and an end-abutting steering angle θ_{abt} acquired when the steering wheel **1** is steered to the right side is sometimes referred to as “right end-abutting steering angle”.

[0195] When, as illustrated in FIG. 13B, both a left end-abutting steering angle θ_{abtl} and a right end-abutting steering angle θ_{abtr} are acquired, the end-abutting detection unit **90** calculates an absolute value of a difference between the end-abutting steering angles θ_{abtl} and θ_{abtr} as the rack stroke approximate value Sta . The end-abutting detection unit **90** compares a minimum value of a rack stroke of a rack shaft that has a possibility of being mistakenly mounted on the vehicle (sometimes refer to as “erroneous stroke minimum value $Sterr$ ”) with the rack stroke approximate value Sta . When the rack stroke approximate value Sta is greater than or equal to the erroneous stroke minimum value $Sterr$, the end-abutting detection unit **90** determines that the end-abutting steering angles θ_{abtl} and θ_{abtr} are valid. When the rack stroke approximate value Sta is less than the erroneous stroke minimum value $Sterr$, the end-abutting detection unit **90** determines that the detection of the end-abutting steering angle θ_{abt} is false detection. The end-abutting detection unit **90** outputs a determination result to the counting unit **92**.

[0196] FIG. 11 is now referred to. The end-abutting steering angle range determination unit **91** determines whether or not variation in right end-abutting steering angles θ_{abt} that the end-abutting detection unit **90** acquired multiple times is less than or equal to a predetermined threshold value θ_{tha3} . Likewise, the end-abutting steering angle range determination unit **91** determines whether or not variation in left end-abutting steering angles θ_{abt} that the end-abutting detection unit **90** acquired multiple times is less than or equal to the predetermined threshold value θ_{tha3} .

[0197] Although a method for calculating variation in right end-abutting steering angles θ_{abt} will be described below with reference to FIGS. 14A to 14C, the same applies to a method for calculating variation in left end-abutting steering angles θ_{abt} .

[0198] FIG. 14A is now referred to. In FIG. 14A, a triangular mark indicates an end-abutting steering angle θ_{abt} that the end-abutting detection unit **90** acquired most recently. Hereinafter, an end-abutting steering angle θ_{abt}

that the end-abutting detection unit **90** acquired most recently is sometimes referred to as “latest end-abutting steering angle”. Circular marks indicate end-abutting steering angles θ_{abt} that the end-abutting detection unit **90** acquired at time points before the end-abutting detection unit **90** acquired the latest end-abutting steering angle θ_{abt} (the triangular mark) and that are stored in an end-abutting steering angle storage unit **94**. Hereinafter, an end-abutting steering angle θ_{abt} that is stored in the end-abutting steering angle storage unit **94** is sometimes referred to as “stored end-abutting steering angle”. The same applies to FIGS. 14B and 14C.

[0199] As illustrated in FIG. 14A, when a difference between a minimum value θ_{min} and a maximum value θ_{max} of the latest end-abutting steering angle θ_{abt} (the triangular mark) and the stored end-abutting steering angles θ_{abt} (the circular marks) is less than or equal to the predetermined threshold value θ_{tha3} , the end-abutting steering angle range determination unit **91** determines that variation in the end-abutting steering angles θ_{abt} is less than or equal to the predetermined threshold value θ_{tha3} . In this case, the end-abutting steering angle range determination unit **91** stores the latest end-abutting steering angle θ_{abt} (the triangular mark) in the end-abutting steering angle storage unit **94** in addition to the stored end-abutting steering angles θ_{abt} (the circular marks).

[0200] FIG. 11 is now referred to. The counting unit **92** counts the number of times that the end-abutting steering angle range determination unit **92** determines that variation in end-abutting steering angles θ_{abt} is less than or equal to the predetermined threshold value θ_{tha3} , every time the end-abutting detection unit **90** acquires an end-abutting steering angle θ_{abt} (a triangular mark). That is, the counting unit **92** counts the number of times of acquisition of an end-abutting steering angle that causes variation in end-abutting steering angles to be less than or equal to the predetermined threshold value θ_{tha3} .

[0201] The counting unit **92** includes a vehicle state determination unit **95**, end-abutting counters **96oL**, **96oR**, **96bL**, and **96bR**, provisional end-abutting counters **97oL**, **97oR**, **97bL**, and **97bR**, and on-center counters **98oL**, **98oR**, **98bL**, and **98bR**.

[0202] In the following description, the end-abutting counters **96oL**, **96oR**, **96bL**, and **96bR** are collectively referred to as “end-abutting counter **96**”, the provisional end-abutting counters **97oL**, **97oR**, **97bL**, and **97bR** are collectively referred to as “provisional end-abutting counter **97**”, and the on-center counters **98oL**, **98oR**, **98bL**, and **98bR** are collectively referred to as “on-center counter **98**”.

[0203] The vehicle state determination unit **95** determines whether the vehicle is in a stop state or a travel state, based on the vehicle speed V_h detected by the vehicle speed sensor **12**. FIG. 15 is an example of a state transition diagram of a vehicle state. When the vehicle speed V_h becomes greater than or equal to a vehicle speed threshold value V_{th} when the vehicle is in the stop state, the vehicle state determination unit **95** determines that the vehicle state has transitioned to the travel state. When the vehicle speed V_h becomes less than the vehicle speed threshold value V_{th} when the vehicle is in the travel state and a count value in the end-abutting counter **96** is “0”, the vehicle state determination unit **95** determines that the vehicle state has transitioned to the stop state. Since the counting unit **92** includes four end-abutting counters **96oL**, **96oR**, **96bL**, and **96bR** as described above,

the vehicle state determination unit **95** may determine whether or not the vehicle state has transitioned to the stop state with respect to each of the counters.

[0204] FIG. **11** is now referred to. The end-abutting counter **96** counts and stores the number of times that the end-abutting steering angle range determination unit **91** determines that variation in end-abutting steering angles θ_{abt} is less than or equal to the predetermined threshold value θ_{tha3} , every time the end-abutting detection unit **90** acquires an end-abutting steering angle θ_{abt} (a triangular mark) when the vehicle state is the travel state.

[0205] That is, when the end-abutting steering angle range determination unit **91** determines that a difference between a minimum value θ_{min} and a maximum value θ_{max} of a latest end-abutting steering angle θ_{abt} (a triangular mark) and stored end-abutting steering angles θ_{abt} (circular marks) is less than or equal to the predetermined threshold value θ_{tha3} when the vehicle state is the travel state, the end-abutting counter **96** increments a count value by one.

[0206] On the other hand, the provisional end-abutting counter **97** counts the number of times that the end-abutting steering angle range determination unit **91** determines that variation in end-abutting steering angles θ_{abt} is less than or equal to the predetermined threshold value θ_{tha3} , every time the end-abutting detection unit **90** acquires an end-abutting steering angle θ_{abt} (a triangular mark) when the vehicle state is the stop state and stores a counted number.

[0207] That is, when the end-abutting steering angle range determination unit **91** determines that a difference between a minimum value θ_{min} and a maximum value θ_{max} of a latest end-abutting steering angle θ_{abt} (a triangular mark) and stored end-abutting steering angles θ_{abt} (circular marks) is less than or equal to the predetermined threshold value θ_{tha3} when the vehicle state is the stop state, the provisional end-abutting counter **97** increments a count value by one.

[0208] The end-abutting counter **96oL** and the provisional end-abutting counter **97oL** are counters that count the number of times that the end-abutting steering angle range determination unit **91** determines that variation in left end-abutting steering angles θ_{abt} is less than or equal to the predetermined threshold value θ_{tha3} when end-abutting occurs in the case of steering to the left side and no end-abutting occurs in the case of steering to the right side.

[0209] In addition, the end-abutting counter **96oR** and the provisional end-abutting counter **97oR** are counters that count the number of times that the end-abutting steering angle range determination unit **91** determines that variation in right end-abutting steering angles θ_{abt} is less than or equal to the predetermined threshold value θ_{tha3} when no end-abutting occurs in the case of steering to the left side and end-abutting occurs in the case of steering to the right side.

[0210] In addition, the end-abutting counter **96bL** and the provisional end-abutting counter **97bL** are counters that count the number of times that the end-abutting steering angle range determination unit **91** determines that variation in left end-abutting steering angles θ_{abt} is less than or equal to the predetermined threshold value θ_{tha3} when end-abutting occurs both in the case of steering to the left side and in the case of steering to the right side.

[0211] In addition, the end-abutting counter **96bR** and the provisional end-abutting counter **97bR** are counters that count the number of times that the end-abutting steering angle range determination unit **91** determines that variation in right end-abutting steering angles θ_{abt} is less than or

equal to the predetermined threshold value θ_{tha3} when end-abutting occurs both in the case of steering to the left side and in the case of steering to the right side.

[0212] When the vehicle state transitions from the stop state to the travel state, the counting unit **92** assigns a count value in the provisional end-abutting counter **97** to a count value in the end-abutting counter **96**. That is, the counting unit **92** replaces the count value in the end-abutting counter **96** by the count value in the provisional end-abutting counter **97**. In addition, the counting unit **92** resets the count value in the provisional end-abutting counter **97** to “0”.

[0213] FIG. **14C** is now referred to. When a latest end-abutting steering angle θ_{abt} (a triangular mark) is closer to a neutral position of the steering mechanism than stored end-abutting steering angles θ_{abt} (circular marks) and a difference between a maximum value θ_{max} (that is, an end-abutting steering angle θ_{abt} farthest from the neutral position) of the stored end-abutting steering angles θ_{abt} (the circular marks) and the latest end-abutting steering angle θ_{abt} (the triangular mark) is greater than the predetermined threshold value θ_{tha3} , the end-abutting steering angle range determination unit **91** determines that variation in the end-abutting steering angles θ_{abt} is not less than or equal to the predetermined threshold value θ_{tha3} . Hereinafter, such a state is sometime referred to as “neutral-side deviation state”.

[0214] When the latest end-abutting steering angle θ_{abt} (the triangular mark) is in the neutral-side deviation state, the end-abutting steering angle range determination unit **91** discards the latest end-abutting steering angle θ_{abt} (the triangular mark).

[0215] In addition, the on-center counter **98** counts the number of times that a latest end-abutting steering angle θ_{abt} (a triangular mark) is determined to be in the neutral-side deviation state and store the counted number. That is, when the end-abutting steering angle range determination unit **91** determines that a latest end-abutting steering angle θ_{abt} (a triangular mark) is in the neutral-side deviation state, the on-center counter **98** increments a count value by one.

[0216] The on-center counter **98oL** is a counter that counts the number of times that the end-abutting steering angle range determination unit **91** determines that a left end-abutting steering angle θ_{abt} is in the neutral-side deviation state when end-abutting occurs in the case of steering to the left side and no end-abutting occurs in the case of steering to the right side.

[0217] The on-center counter **98oR** is a counter that counts the number of times that the end-abutting steering angle range determination unit **91** determines that a right end-abutting steering angle θ_{abt} is in the neutral-side deviation state when no end-abutting occurs in the case of steering to the left side and end-abutting occurs in the case of steering to the right side.

[0218] The on-center counter **98bL** is a counter that counts the number of times that the end-abutting steering angle range determination unit **91** determines that a left end-abutting steering angle θ_{abt} is in the neutral-side deviation state when end-abutting occurs both in the case of steering to the left side and in the case of steering to the right side.

[0219] The on-center counter **98bR** is a counter that counts the number of times that the end-abutting steering angle range determination unit **91** determines that a right end-abutting steering angle θ_{abt} is in the neutral-side deviation

state when end-abutting occurs both in the case of steering to the left side and in the case of steering to the right side.

[0220] When a count value in one of the on-center counters 98oL, 98oR, 98bL, and 98bR becomes greater than or equal to a predetermined threshold value, the end-abutting steering angle range determination unit 91 deletes left or right end-abutting steering angles θ_{abt} (circular marks) stored in the end-abutting steering angle storage unit 94. In addition, the counting unit 92 resets the count values in the end-abutting counters 96oL, 96oR, 96bL, and 96bR, the provisional end-abutting counters 97oL, 97oR, 97bL, and 97bR, and the on-center counters 98oL, 98oR, 98bL, and 98bR to “0”.

[0221] FIG. 14B is now referred to. When an end-abutting steering angle θ_{abt} (a triangular mark) acquired most recently is located farther from the neutral position of the steering mechanism than stored end-abutting steering angles θ_{abt} (circular marks) and a difference between a minimum value θ_{min} (that is, an end-abutting steering angle θ_{abt} closest to the neutral position) of the stored end-abutting steering angles θ_{abt} (the circular marks) and the end-abutting steering angle θ_{abt} (the triangular mark) acquired most recently is greater than the predetermined threshold value θ_{tha3} , the end-abutting steering angle range determination unit 91 determines that variation in the end-abutting steering angles θ_{abt} is not less than or equal to the predetermined threshold value θ_{tha3} . Hereinafter, such a state is sometime referred to as “end-side deviation state”.

[0222] When a latest end-abutting steering angle θ_{abt} (a triangular mark) is in the end-side deviation state, the end-abutting steering angle range determination unit 91 deletes end-abutting steering angles θ_{abt} (circular marks) stored at time points before the latest end-abutting steering angle θ_{abt} (the triangular mark) is acquired from the end-abutting steering angle storage unit 94 and stores the latest end-abutting steering angle θ_{abt} (the triangular mark) in the end-abutting steering angle storage unit 94.

[0223] In addition, the counting unit 92 resets count values in the end-abutting counter 96, the provisional end-abutting counter 97, and the on-center counter 98 to “0”. When the vehicle state is the travel state, the counting unit 92 increments the end-abutting counter 96 by one, and when the vehicle state is the stop state, the counting unit 92 increments the provisional end-abutting counter 97 by one.

[0224] Specifically, when end-abutting occurs in the case of steering to the left side and no end-abutting occurs in the case of steering to the right side and the latest end-abutting steering angle θ_{abt} (the triangular mark) is a left end-abutting steering angle θ_{abt} , the end-abutting steering angle range determination unit 91 deletes stored left end-abutting steering angles θ_{abt} (circular marks) from the end-abutting steering angle storage unit 94 and stores the latest left end-abutting steering angle θ_{abt} (the triangular mark) in the end-abutting steering angle storage unit 94. In addition, the counting unit 92 resets the count values in the end-abutting counter 96oL, the provisional end-abutting counter 97oL, and the on-center counter 98oL to “0”. When the vehicle state is the travel state, the counting unit 92 increments the end-abutting counter 96oL by one, and when the vehicle state is the stop state, the counting unit 92 increments the provisional end-abutting counter 97oL by one.

[0225] In addition, when no end-abutting occurs in the case of steering to the left side and end-abutting occurs in the case of steering to the right side and the latest end-abutting

steering angle θ_{abt} (the triangular mark) is a right end-abutting steering angle θ_{abt} , the end-abutting steering angle range determination unit 91 deletes stored right end-abutting steering angles θ_{abt} (circular marks) from the end-abutting steering angle storage unit 94 and stores the latest right end-abutting steering angle θ_{abt} (the triangular mark) in the end-abutting steering angle storage unit 94. In addition, the counting unit 92 resets the count values in the end-abutting counter 96oR, the provisional end-abutting counter 97oR, and the on-center counter 98oR to “0”. When the vehicle state is the travel state, the counting unit 92 increments the end-abutting counter 96oR by one, and when the vehicle state is the stop state, the counting unit 92 increments the provisional end-abutting counter 97oR by one.

[0226] In addition, when end-abutting occurs both in the case of steering to the left side and in the case of steering to the right side and the latest end-abutting steering angle θ_{abt} (the triangular mark) is a left end-abutting steering angle θ_{abt} , the end-abutting steering angle range determination unit 91 deletes stored left end-abutting steering angles θ_{abt} (circular marks) from the end-abutting steering angle storage unit 94 and stores the latest left end-abutting steering angle θ_{abt} (the triangular mark) in the end-abutting steering angle storage unit 94. In addition, the counting unit 92 resets the count values in the end-abutting counter 96bL, the provisional end-abutting counter 97bL, and the on-center counter 98bL to “0”. When the vehicle state is the travel state, the counting unit 92 increments the end-abutting counter 96bL by one, and when the vehicle state is the stop state, the counting unit 92 increments the provisional end-abutting counter 97bL by one.

[0227] In addition, when end-abutting occurs both in the case of steering to the left side and in the case of steering to the right side and the latest end-abutting steering angle θ_{abt} (the triangular mark) is a right end-abutting steering angle θ_{abt} , the end-abutting steering angle range determination unit 91 deletes stored right end-abutting steering angles θ_{abt} (circular marks) from the end-abutting steering angle storage unit 94 and stores the latest right end-abutting steering angle θ_{abt} (the triangular mark) in the end-abutting steering angle storage unit 94. In addition, the counting unit 92 resets the count values in the end-abutting counter 96bR, the provisional end-abutting counter 97bR, and the on-center counter 98bR to “0”. When the vehicle state is the travel state, the counting unit 92 increments the end-abutting counter 96bR by one, and when the vehicle state is the stop state, the counting unit 92 increments the provisional end-abutting counter 97bR by one.

[0228] When end-abutting occurs both in the case of steering to the left side and in the case of steering to the right side, the counting unit 92 receives a determination result of comparison between a rack stroke approximate value Sta (FIG. 13B) that the end-abutting detection unit 90 calculates, as an absolute value of a difference between the left and right end-abutting steering angles θ_{abt} and the erroneous stroke minimum value $Sterr$, from the end-abutting detection unit 90.

[0229] When the rack stroke approximate value Sta is less than the erroneous stroke minimum value $Sterr$, the counting unit 92 determines that false detection of an end-abutting steering angle θ_{abt} has occurred and saves the count values in the end-abutting counters 96bL and 96bR in the provisional end-abutting counters 97bL and 97bR. That is, the counting unit 92 replaces the count values in the provisional

end-abutting counters 97bL and 97bR by the count values in the end-abutting counters 96bL and 96bR, respectively and resets the count values in the end-abutting counters 96bL and 96bR to “0”.

[0230] Subsequently, when the vehicle state transitions to the travel state in the determination processing of the vehicle state and the detection of the end-abutting steering angle θ_{abt} is determined to be reliable, the count values in the provisional end-abutting counters 97bL and 97bR are returned to the end-abutting counters 96bL and 96bR, respectively.

[0231] Note that variation in a latest end-abutting steering angle θ_{abt} (a triangular mark) and stored end-abutting steering angles θ_{abt} (circular marks) being less than or equal to the predetermined threshold value θ_{tha3} as illustrated in FIG. 14A is sometimes referred to as “an end-abutting steering angle θ_{abt} acquired most recently falls within the predetermined variation range θ_{tha3} ” in the following description. In addition, a latest end-abutting steering angle θ_{abt} (a triangular mark) being in the end-side deviation state or the neutral-side deviation state as illustrated in FIG. 14B or FIG. 14C is sometimes referred to as “an end-abutting steering angle θ_{abt} acquired most recently falls outside the predetermined variation range θ_{tha3} ” in the following description.

[0232] FIG. 11 is now referred to. The relearning unit 93 determines necessity of relearning of one of the virtual rack end positions θ_{ev} , based on a count value in the end-abutting counter 96, and when the relearning of the virtual rack end position θ_{ev} is to be performed, the relearning unit 93 outputs a relearning command signal Cmd to the terminal position learning unit 46.

[0233] For example, when a count value in the end-abutting counter 96 exceeds a predetermined threshold value, the relearning unit 93 determines that relearning of a corresponding virtual rack end position θ_{ev} is necessary and outputs a relearning command signal Cmd.

[0234] For example, when end-abutting occurs in the case of steering to the left side and no end-abutting occurs in the case of steering to the right side and the count value in the end-abutting counter 96oL for the left side exceeds the predetermined threshold value, the relearning unit 93 may output a relearning command signal Cmd commanding relearning of the left virtual rack end position θ_{evl} to the terminal position learning unit 46. The terminal position learning unit 46 resets the left virtual rack end position θ_{evl} to the initial value $-\theta_{int}$ in accordance with the relearning command signal Cmd. Because of this configuration, the terminal position learning unit 46 relearns the left virtual rack end position θ_{evl} .

[0235] In addition, for example, when no end-abutting occurs in the case of steering to the left side and end-abutting occurs in the case of steering to the right side and the count value in the end-abutting counter 96oR for the right side exceeds the predetermined threshold value, the relearning unit 93 may output a relearning command signal Cmd commanding relearning of the right virtual rack end position θ_{evr} to the terminal position learning unit 46. The terminal position learning unit 46 resets the right virtual rack end position θ_{evr} to the initial value θ_{int} in accordance with the relearning command signal Cmd. Because of this configuration, the terminal position learning unit 46 relearns the right virtual rack end position θ_{evr} .

[0236] In addition, for example, when end-abutting occurs both in the case of steering to the left side and in the case of steering to the right side and a sum of the count value in the end-abutting counter 96bL for the left side and the count value in the end-abutting counter 96bR for the right side is greater than or equal to a predetermined threshold value, the relearning unit 93 calculates the rack stroke minimum value St_{min} , the rack stroke maximum value St_{max} , the learning threshold value θ_{lth} , the rack end maximum value θ_{evmax} , and the initial value θ_{int} , which are set values used for the learning of the virtual rack end positions θ_{ev} , based on end-abutting steering angles θ_{abt} stored in the end-abutting steering angle storage unit 94 of the end-abutting steering angle range determination unit 91.

[0237] With reference to FIG. 16, an example of a method for calculating the rack stroke minimum value St_{min} , the rack stroke maximum value St_{max} , the learning threshold value θ_{lth} , the rack end maximum value θ_{evmax} , and the initial value θ_{int} will be described. Note that in FIG. 16, a reference sign “RC” denotes a center position between the left and right rack ends, and “0 [deg]” denotes a position at which the steering angle θ_h detected by the steering angle sensor 14 is “0”.

[0238] The relearning unit 93 calculates a sum of an absolute value of a maximum value of left end-abutting steering angles θ_{abt} (that is, a left end-abutting steering angle θ_{abt} among the left end-abutting steering angles θ_{abt} that is closest to the neutral position) and an absolute value of a minimum value of right end-abutting steering angles θ_{abt} (that is, a right end-abutting steering angle θ_{abt} among the right end-abutting steering angles θ_{abt} that is closest to the neutral position) as a provisional stroke $St_p = |\max(\text{left end-abutting steering angles } \theta_{abt})| + |\min(\text{right end-abutting steering angles } \theta_{abt})|$.

[0239] Next, the relearning unit 93 calculates a half of an absolute value of a difference between the absolute value of the maximum value of the left end-abutting steering angles θ_{abt} and the absolute value of the minimum value of the right end-abutting steering angles θ_{abt} as a provisional assembly error $e\theta = |\max(\text{left end-abutting steering angles } \theta_{abt})| - |\min(\text{right end-abutting steering angles } \theta_{abt})|/2$.

[0240] The relearning unit 93 calculates a subtraction result obtained by subtracting predetermined thermal expansion/contraction error of a manual steering gear and learning error from the provisional stroke St_p as a rack stroke minimum value $St_{min} = St_p - (\text{thermal expansion/contraction error of manual steering gear}) - (\text{learning error})$.

[0241] The relearning unit 93 calculates an addition result obtained by adding the predetermined thermal expansion/contraction error of the manual steering gear, the learning error, and sensor error to the provisional stroke St_p as a rack stroke maximum value $St_{max} = St_p + (\text{thermal expansion/contraction error of manual steering gear}) + (\text{learning error}) + (\text{sensor error})$.

[0242] The relearning unit 93 calculates a sum of a half of the rack stroke maximum value St_{max} and the provisional assembly error $e\theta$ as a rack end maximum value $\theta_{evmax} = (St_{max}/2 + e\theta)$.

[0243] The relearning unit 93 calculates a subtraction result obtained by subtracting the rack end maximum value θ_{evmax} from the rack stroke minimum value St_{min} as an initial value $\theta_{int} = (St_{min} - \theta_{evmax})$.

[0244] The relearning unit 93 calculates a subtraction result obtained by subtracting the provisional assembly error

$e\theta$ from a half of the rack stroke minimum value $Stmin$ as a learning threshold value $\theta_{lth}=Stmin/2-(\text{provisional assembly error } e\theta)$.

[0245] The relearning unit 93 outputs the calculated initial value θ_{int} and rack stroke maximum value $Stmax$ to the terminal position learning unit 46. In addition, the relearning unit 93 outputs the calculated rack stroke maximum value $Stmax$, rack stroke minimum value $Stmin$, and learning threshold value θ_{lth} to the learning state determination unit 51. The relearning unit 93 may be configured to be able to output the rack end maximum value θ_{evmax} .

[0246] The terminal position learning unit 46 updates the initial values θ_{int} of the virtual rack end positions θ_{evr} and θ_{evl} at the time of learning start to values that the relearning unit 93 recalculated. The terminal position learning unit 46 updates the rack stroke maximum value $Stmax$ to be used for determination of offset error to a value that the relearning unit 93 recalculated.

[0247] In addition, the learning state determination unit 51 updates the rack stroke maximum value $Stmax$, the rack stroke minimum value $Stmin$, and the learning threshold value θ_{lth} to be used for determination of a learning state to values that the relearning unit 93 recalculated.

[0248] The relearning unit 93 may output a relearning command signal Cmd commanding relearning of the left virtual rack end position θ_{evl} and the right virtual rack end position θ_{evr} to the terminal position learning unit 46. The terminal position learning unit 46 resets the left virtual rack end position θ_{evl} to the initial value $-\theta_{int}$ and resets the right virtual rack end position θ_{evr} to the initial value θ_{int} in accordance with the relearning command signal Cmd . Because of this configuration, the terminal position learning unit 46 relearns the left virtual rack end position θ_{evl} and the right virtual rack end position θ_{evr} .

(Operation)

[0249] FIG. 17 is a flowchart of an example of a turning control method of the embodiment.

[0250] In step S1, the end-abutting detection unit 90 retains a steering angle θ_{dt} when the steering torque Th is greater than or equal to a predetermined threshold value $Tth1$.

[0251] In step S2, the end-abutting detection unit 90 determines whether or not an absolute value $|\theta_{ev}-\theta_{dt}|$ of a difference between the retained steering angle θ_{dt} and one of the virtual rack end positions θ_{ev} is greater than the steering angle threshold value θ_{tha1} . When the absolute value $|\theta_{ev}-\theta_{dt}|$ is not greater than the steering angle threshold value θ_{tha1} (step S2: N), the process returns to step S1. When the absolute value $|\theta_{ev}-\theta_{dt}|$ is greater than the steering angle threshold value θ_{tha1} (step S2: Y), the process proceeds to step S3.

[0252] In step S3, the end-abutting detection unit 90 retains an end-abutting steering angle candidate θ_{cd} when, after the steering torque Th has become greater than or equal to the predetermined threshold value $Tth1$, the column output shaft torque Tc becomes less than the predetermined threshold value $Tth2$ and the steering torque Th becomes less than the predetermined threshold value $Tth1$.

[0253] In step S4, the end-abutting detection unit 90 determines whether or not an absolute value $|\theta_{ev}-\theta_{cd}|$ of a difference between an end-abutting steering angle candidate θ_{cd} and the virtual rack end position θ_{ev} is greater than the steering angle threshold value θ_{tha2} . When the absolute

value $|\theta_{ev}-\theta_{cd}|$ is not greater than the steering angle threshold value θ_{tha2} (step S4: N), the process returns to step S1. When the absolute value $|\theta_{ev}-\theta_{cd}|$ is greater than the steering angle threshold value θ_{tha2} (step S4: Y), the process proceeds to step S5.

[0254] In step S5, the vehicle state determination unit 95 determines whether the vehicle is in the stop state or the travel state. When the vehicle state transitions from the stop state to the travel state, the counting unit 92 assigns a count value in the provisional end-abutting counter 97 to count value in the end-abutting counter 96. In addition, the counting unit 92 resets the count value in the provisional end-abutting counter 97 to "0".

[0255] In step S6, the end-abutting detection unit 90 determines whether or not end-abutting occurs both in the case of the steering wheel 1 being steered to the left side and in the case of being steered to the right side. When end-abutting occurs both in the case of steering to the left side and in the case of steering to the right side (step S6: Y), the process proceeds to step S8. When end-abutting occurs only either in the case of steering to the left side or in the case of steering to the right side (step S6: N), the process proceeds to step S7.

[0256] In step S7, the relearning determination unit 53 performs one-side end-abutting processing. Details of the one-side end-abutting processing will be described later with reference to FIG. 18. Subsequently, the process proceeds to step S9.

[0257] In step S8, the relearning determination unit 53 performs both-side end-abutting processing. Details of the both-side end-abutting processing will be described later with reference to FIG. 19. Subsequently, the process proceeds to step S9.

[0258] In step S9, the controller 30 determines whether or not the IGN key 11 is turned off. When the IGN key 11 is not turned off (step S9: N), the process returns to step S1. When the IGN key 11 is turned off (step S9: Y), the process terminates.

[0259] FIG. 18 is a flowchart of an example of the one-side end-abutting processing (S7) in FIG. 17. The one-side end-abutting processing is performed independently when end-abutting occurs in the case of steering to the right side (that is, a right end-abutting steering angle θ_{abt} is acquired) and when end-abutting occurs in the case of steering to the left side (that is, a left end-abutting steering angle θ_{abt} is acquired).

[0260] In step S10, the end-abutting detection unit 90 calculates a rack stroke approximate value Sta , based on an end-abutting steering angle θ_{abt} acquired most recently. For example, when a right end-abutting steering angle θ_{abt} is acquired, the end-abutting detection unit 90 calculates an absolute value of a difference between the left virtual rack end position θ_{evl} and the right end-abutting steering angle θ_{abt} as the rack stroke approximate value Sta . When a left end-abutting steering angle θ_{abt} is acquired, the end-abutting detection unit 90 calculates an absolute value of a difference between the right virtual rack end position θ_{evr} and the left end-abutting steering angle θ_{abt} as the rack stroke approximate value Sta . When error between the rack stroke approximate value Sta and a predetermined value does not fall within a threshold value (step S10: N), the one-side end-abutting processing is terminated. When the error between the rack stroke approximate value Sta and the

predetermined value falls within the threshold value (step S10: Y), the process proceeds to step S11.

[0261] In step S11, the end-abutting steering angle range determination unit 91 determines whether or not an end-abutting steering angle θ_{abt} acquired most recently falls within a predetermined variation range θ_{tha3} . When the end-abutting steering angle θ_{abt} acquired most recently falls outside the predetermined variation range θ_{tha3} (step S11: N), the process proceeds to step S15. When the end-abutting steering angle θ_{abt} acquired most recently falls within the predetermined variation range θ_{tha3} (step S11: Y), the process proceeds to step S12.

[0262] In step S12, the counting unit 92 increments a count value in the end-abutting counter 96 or the provisional end-abutting counter 97 by one. Specifically, when the vehicle is in the travel state and a left end-abutting steering angle θ_{abt} is acquired, the counting unit 92 increments a count value in the end-abutting counter 96oL by one. When the vehicle is in the travel state and a right end-abutting steering angle θ_{abt} is acquired, the counting unit 92 increments a count value in the end-abutting counter 96oR by one. When the vehicle is in the stop state and a left end-abutting steering angle θ_{abt} is acquired, the counting unit 92 increments a count value in the end-abutting counter 97oL by one. When the vehicle is in the stop state and a right end-abutting steering angle θ_{abt} is acquired, the counting unit 92 increments a count value in the end-abutting counter 97oR by one.

[0263] In step S13, the relearning unit 93 determines whether or not a count value in the end-abutting counters 96 exceeds a predetermined threshold value. When a count value in the end-abutting counter 96 does not exceed the predetermined threshold value (step S13: N), the one-side end-abutting processing is terminated. When a count value in the end-abutting counter 96 exceeds the predetermined threshold value (step S13: Y), the process proceeds to step S14.

[0264] In step S14, the relearning unit 93 outputs a relearning command signal Cmd to the terminal position learning unit 46.

[0265] Specifically, when the count value in the left end-abutting counter 96oL exceeds the predetermined threshold value, the relearning unit 93 outputs a relearning command signal Cmd commanding relearning of the left virtual rack end position θ_{evl} . The terminal position learning unit 46 resets the left virtual rack end position θ_{evl} to an initial value-fint in accordance with the relearning command signal Cmd. When the count value in the right end-abutting counter 96oR exceeds the predetermined threshold value, the relearning unit 93 outputs a relearning command signal Cmd commanding relearning of the right virtual rack end position θ_{evr} . The terminal position learning unit 46 resets the right virtual rack end position θ_{evr} to an initial value-fint in accordance with the relearning command signal Cmd. Subsequently, the one-side end-abutting processing is terminated.

[0266] In step S15, the end-abutting steering angle range determination unit 91 determines whether or not the end-abutting steering angle θ_{abt} acquired most recently is in the neutral-side deviation state. When the end-abutting steering angle θ_{abt} acquired most recently is in the end-side deviation state (step S15: N), the process proceeds to step S19. When the end-abutting steering angle θ_{abt} acquired most

recently is in the neutral-side deviation state (step S15: Y), the process proceeds to step S16.

[0267] In step S16, the counting unit 92 increments a count value in the on-center counter 98 by one. Specifically, when the end-abutting steering angle θ_{abt} acquired most recently is a left end-abutting steering angle θ_{abt} , the counting unit 92 increments the count value in the on-center counter 98oL by one, and when the end-abutting steering angle θ_{abt} acquired most recently is a right end-abutting steering angle θ_{abt} , the counting unit 92 increments the count value in the on-center counter 98oR by one.

[0268] In step S17, the counting unit 92 determines whether or not the count value in either the on-center counters 98oL or 98oR is greater than or equal to the predetermined threshold value. When neither count value is greater than or equal to the threshold value (step S17: N), the one-side end-abutting processing is terminated. When the count value in either the end-abutting counters 98oL or 98oR is greater than or equal to the threshold value (step S17: Y), the process proceeds to step S18.

[0269] In step S18, the counting unit 92 resets the count values in the end-abutting counters 96oL, 96oR, 96bL, and 96bR, the provisional end-abutting counters 97oL, 97oR, 97bL, and 97bR, and the on-center counters 98oL, 98oR, 98bL, and 98bR to "0". In addition, the end-abutting steering angle range determination unit 91 deletes the left or right end-abutting steering angles θ_{abt} stored in the end-abutting steering angle storage unit 94. Subsequently, the one-side end-abutting processing is terminated.

[0270] In step S19, the counting unit 92 resets the end-abutting counter 96oL and the provisional end-abutting counter 97oL or the end-abutting counter 96oR and the provisional end-abutting counter 97oR to "0", depending on which one of left and right end-abutting steering angles θ_{abt} is in the end-side deviation state. In addition, the end-abutting steering angle range determination unit 91 deletes one of the left and right end-abutting steering angles θ_{abt} stored in the end-abutting steering angle storage unit 94. Specifically, when a left end-abutting steering angle θ_{abt} is in the end-side deviation state, the counting unit 92 resets the end-abutting counter 96oL and the provisional end-abutting counter 97oL to "0" and the end-abutting steering angle range determination unit 91 deletes the left end-abutting steering angle θ_{abt} . When a right end-abutting steering angle θ_{abt} is in the end-side deviation state, the counting unit 92 resets the end-abutting counter 96oR and the provisional end-abutting counter 97oR to "0" and the end-abutting steering angle range determination unit 91 deletes the right end-abutting steering angle θ_{abt} .

[0271] Processing in step S20 is the same as the processing in step S12. Subsequently, the one-side end-abutting processing is terminated.

[0272] FIG. 19 is a flowchart of an example of the both-side end-abutting processing (S8) in FIG. 17.

[0273] In step S30, the end-abutting steering angle range determination unit 91 determines whether or not an end-abutting steering angle θ_{abt} acquired most recently falls within a predetermined variation range θ_{tha3} . When the end-abutting steering angle θ_{abt} acquired most recently falls outside the predetermined variation range θ_{tha3} (step S30: N), the process proceeds to step S36. Note that processing in steps S36 to S42 is performed independently when end-abutting occurs in the case of steering to the right side (that is, a right end-abutting steering angle θ_{abt} is acquired) and

when end-abutting occurs in the case of steering to the left side (that is, a left end-abutting steering angle θ_{abt} is acquired).

[0274] In contrast, when the end-abutting steering angle θ_{abt} acquired most recently falls within the predetermined variation range θ_{tha3} (step S30: Y), the process proceeds to step S31.

[0275] In step S31, the counting unit 92 increments a count value in the end-abutting counter 96 or the provisional end-abutting counter 97 by one. Specifically, when the vehicle is in the travel state and a left end-abutting steering angle θ_{abt} is acquired, the counting unit 92 increments a count value in the end-abutting counter 96bL by one. When the vehicle is in the travel state and a right end-abutting steering angle θ_{abt} is acquired, the counting unit 92 increments a count value in the end-abutting counter 96bR by one. Specifically, when the vehicle is in the stop state and a left end-abutting steering angle θ_{abt} is acquired, the counting unit 92 increments a count value in the end-abutting counter 97bL by one. When the vehicle is in the stop state and a right end-abutting steering angle θ_{abt} is acquired, the counting unit 92 increments a count value in the end-abutting counter 97bR by one.

[0276] In step S32, the end-abutting detection unit 90 calculates an absolute value of a difference between the left and right end-abutting steering angles θ_{abt} as a rack stroke approximate value Sta . For example, the end-abutting detection unit 90 may calculate an absolute value of a difference between a minimum value of stored right end-abutting steering angles θ_{abt} (that is, a right end-abutting steering angle θ_{abt} among the stored right end-abutting steering angles θ_{abt} that is closest to the neutral position) and a maximum value of stored left end-abutting steering angles θ_{abt} (that is, a left end-abutting steering angle θ_{abt} among the stored left end-abutting steering angles θ_{abt} that is closest to the neutral position) as a rack stroke approximate value $Sta = |\min(\text{right end-abutting steering angles } \theta_{abt}) - \max(\text{left end-abutting steering angles } \theta_{abt})|$. When the rack stroke approximate value Sta is not greater than or equal to the erroneous stroke minimum value $Sterr$ (step S32: N), the process proceeds to step S35. When the rack stroke approximate value Sta is greater than or equal to the erroneous stroke minimum value $Sterr$ (step S32: Y), the process proceeds to step S33.

[0277] In step S33, the relearning unit 93 determines whether or not a sum of the count value in the end-abutting counter 96bL and the count value in the end-abutting counter 96bR exceeds a predetermined threshold value. When the sum of the count values is not greater than or equal to the threshold value (step S33: N), the both-side end-abutting processing is terminated. When the sum of the count values is greater than or equal to the threshold value (step S33: Y), the process proceeds to step S34.

[0278] In step S34, the relearning unit 93 calculates setting values to be used for learning of the virtual rack end positions θ_{ev} (the rack stroke minimum value $Stmin$, the rack stroke maximum value $Stmax$, the learning threshold value θ_{lth} , the rack end maximum value θ_{evmax} , and the initial value θ_{int}). The relearning unit 93 outputs the calculated initial value θ_{int} and rack stroke maximum value $Stmax$ to the terminal position learning unit 46 and outputs the calculated rack stroke maximum value $Stmax$, rack stroke minimum value $Stmin$, and learning threshold value θ_{lth} to the learning state determination unit 51. The terminal

position learning unit 46 and the learning state determination unit 51 update the setting values to values received from the relearning unit 93.

[0279] The relearning unit 93 outputs a relearning command signal Cmd to the terminal position learning unit 46. The terminal position learning unit 46 resets the left virtual rack end position θ_{evl} to the initial value θ_{int} and resets the right virtual rack end position θ_{evr} to the initial value θ_{int} in accordance with the relearning command signal Cmd . Subsequently, the both-side end-abutting processing is terminated.

[0280] In step S35, the counting unit 92 saves the count values in the end-abutting counters 96bL and 96bR in the provisional end-abutting counters 97bL and 97bR. Subsequently, the both-side end-abutting processing is terminated.

[0281] In step S36, the end-abutting detection unit 90 calculates an absolute value of a difference between the left and right end-abutting steering angles θ_{abt} as a rack stroke approximate value Sta . When the rack stroke approximate value Sta is not greater than or equal to the erroneous stroke minimum value $Sterr$ (step S36: N), the both-side end-abutting processing is terminated. When the rack stroke approximate value Sta is greater than or equal to the erroneous stroke minimum value $Sterr$ (step S36: Y), the process proceeds to step S37.

[0282] In step S37, the end-abutting steering angle range determination unit 91 determines whether or not the end-abutting steering angle θ_{abt} acquired most recently is in the neutral-side deviation state. When the end-abutting steering angle θ_{abt} acquired most recently is in the end-side deviation state (step S37: N), the process proceeds to step S41. When the end-abutting steering angle θ_{abt} acquired most recently is in the neutral-side deviation state (step S37: Y), the process proceeds to step S38.

[0283] In step S38, the counting unit 92 increments a count value in the on-center counter 98 by one. Specifically, when the end-abutting steering angle θ_{abt} acquired most recently is a left end-abutting steering angle θ_{abt} , the counting unit 92 increments the count value in the on-center counter 98bL by one, and when the end-abutting steering angle θ_{abt} acquired most recently is a right end-abutting steering angle θ_{abt} , the counting unit 92 increments the count value in the on-center counter 98bR by one.

[0284] In step S39, the counting unit 92 determines whether or not the count value in either the on-center counters 98bL or 98bR is greater than or equal to the predetermined threshold value. When neither count value is greater than or equal to the threshold value (step S39: N), the both-side end-abutting processing is terminated. When the count value in either the end-abutting counters 98bL or 98bR is greater than or equal to the threshold value (step S39: Y), the process proceeds to step S40.

[0285] Processing in step S40 is the same as the processing in step S18 in FIG. 18. Subsequently, the both-side end-abutting processing is terminated.

[0286] In step S41, the counting unit 92 resets either the end-abutting counter 96bL and the provisional end-abutting counter 97bL or the end-abutting counter 96bR and the provisional end-abutting counter 97bR to "0", depending on which one of the left and right end-abutting steering angles θ_{abt} is in the end-side deviation state. In addition, the end-abutting steering angle range determination unit 91 deletes one of the left and right end-abutting steering angles θ_{abt} stored in the end-abutting steering angle storage unit

94. Specifically, when the left end-abutting steering angle θ_{abt} is in the end-side deviation state, the counting unit **92** resets the end-abutting counter **96bL** and the provisional end-abutting counter **97bL** to “0” and the end-abutting steering angle range determination unit **91** deletes the left end-abutting steering angle θ_{abt} . When the right end-abutting steering angle θ_{abt} is in the end-side deviation state, the counting unit **92** resets the end-abutting counter **96bR** and the provisional end-abutting counter **97bR** to “0” and the end-abutting steering angle range determination unit **91** deletes the right end-abutting steering angle θ_{abt} .

[0287] Processing in step **S42** is the same as the processing in step **S31**. Subsequently, the both-side end-abutting processing is terminated.

(Variations)

[0288] Although an embodiment in which the turning control device of the present invention is applied to the electric power steering device is described above, the turning control device of the present invention is widely applicable to a variety of turning control devices other than the electric power steering device as long as the turning control device is a turning control device that generates force to turn the steered wheels of a vehicle by an actuator. For example, the turning control device of the present invention may be applied to a steering device of a steer-by-wire (SBW) type in which the steering wheel and the steered wheels are mechanically separated from each other. In this case, the steering torque T_h does not have to be added to the motor torque when the column output shaft torque T_c is calculated.

[0289] (1) The end-abutting detection unit **90** may correct an end-abutting steering angle θ_{abt} according to the amount of deformation of a mechanical part due to the column output shaft torque T_c . This configuration enables influence of torsion of the turning mechanism to be reduced. The amount of deformation can be calculated from the column output shaft torque T_c and rigidity (spring constant) of the mechanical part. The rigidity is a ratio of a change amount of the column output shaft torque T_c to a change amount of the steering angle θ_h at a point indicated by the reference sign θ_1 or θ_2 in FIGS. 7A and 7B and may be acquired through an experiment or the like.

[0290] Characteristics illustrated in FIGS. 7A and 7B changes due to viscous resistance, which depends on steering velocity. There is a possibility that the steering velocity influences a learned value of a rack end. It may be configured such that a change amount of the learned value of the rack end with respect to a change amount of the steering velocity is acquired through an experiment or the like and stored as steering velocity-correction amount characteristics and the end-abutting steering angle θ_{abt} is corrected based on the steering velocity and the steering velocity-correction amount characteristics.

[0291] (2) In the above-described embodiment, when the rack stroke St between the right and left virtual rack end positions θ_{ev} exceeds the rack stroke maximum value St_{max} , a virtual rack end position θ_{ev} of a stroke end where no end-abutting is detected is reset in such a way that the rack stroke St coincides with the predetermined rack stroke minimum value. In place of the configuration, when one of the right and left virtual rack end positions θ_{ev} exceeds the rack end maximum value θ_{evmax} , the other of the right and

left virtual rack end positions θ_{ev} may be reset in such a way that the rack stroke St coincides with the predetermined rack stroke minimum value.

Advantageous Effects of Embodiment

[0292] (1) A turning control device includes: a position detection unit configured to detect a steered position of a turning mechanism of a vehicle; a terminal position learning unit configured to learn a terminal position of the turning mechanism, based on the steered position detected by the position detection unit; and a relearning determination unit configured to determine necessity of relearning of the terminal position. The relearning determination unit includes: an end-abutting detection unit configured to detect occurrence of end-abutting, the end-abutting being a state in which the turning mechanism is turned to the terminal position, and acquire an end-abutting steering angle, the end-abutting steering angle being a steering angle when occurrence of end-abutting is detected; an end-abutting steering angle range determination unit configured to determine whether or not variation in the end-abutting steering angles acquired multiple times is less than or equal to a predetermined threshold value; and a relearning unit configured to, when the variation is less than or equal to the predetermined threshold value, reset the learned terminal position to an initial value.

[0293] Because of this configuration, it is possible to prevent mistakenly detecting end-abutting due to rapid turning-back steering, collision of a tire with a curb, or the like before end-abutting actually occurs and relearning a terminal position. Thus, incorrect learning of a terminal position can be prevented.

[0294] (2) When occurrence of end-abutting is detected both in a case of steering to a left side and in a case of steering to a right side and each of variation in the end-abutting steering angles in the case of steering to the left side and variation in the end-abutting steering angles in the case of steering to the right side is less than or equal to the predetermined threshold value, the relearning unit may calculate estimated rack stroke, based on the end-abutting steering angles and reset the initial value, based on the estimated rack stroke.

[0295] Because of this configuration, when a rack shaft having a length different from length of a rack shaft that should be originally mounted is mistakenly mounted, a learning initial value can be set according to rack length of the rack shaft that is actually mounted.

[0296] (3) The turning control device may include a counting unit configured to count a number of times of acquisition of the end-abutting steering angles having variation less than or equal to the predetermined threshold value, and when the number of times of acquisition is greater than or equal to a first threshold number of times, the relearning unit may reset the learned terminal position to the initial value.

[0297] Because of this configuration, whether or not variation in the end-abutting steering angles is less than or equal to a predetermined threshold value can be determined more accurately.

[0298] (4) The steering angle range determination unit may store the acquired end-abutting steering angle and determine whether or not variation in the end-abutting steering angles is less than or equal to the predetermined threshold value, based on a difference between the stored end-abutting steering angle and a steering angle when occur-

rence of end-abutting is detected. The counting unit may reset counting of the number of times of acquisition when variation in the end-abutting steering angles is determined not to be less than or equal to the predetermined threshold value.

[0299] Because of this configuration, mistakenly relearning a terminal position when variation in the end-abutting steering angles is temporarily less than or equal to a predetermined threshold value can be prevented.

[0300] (5) The counting unit may count a number of times that it is determined that a second steering angle, the second steering angle being a steering angle when occurrence of end-abutting is detected, is closer to a neutral position of the turning mechanism than a first steering angle, the first steering angle being a steering angle located farthest from the neutral position among the stored end-abutting steering angles, and a difference between the first steering angle and the second steering angle is not less than or equal to the predetermined threshold value, as a number of times of being on-center, and when the number of times of being on-center is greater than or equal to a second threshold number of times, reset counting of the number of times of acquisition.

[0301] Because of this configuration, mistakenly relearning a terminal position while the steering wheel cannot be steered to an angle at which end-abutting actually occurs because a tire is in contact with a curb or the like can be prevented.

[0302] (6) The counting unit may include a count storage unit configured to retain a number of times that the end-abutting steering angle having variation less than or equal to the predetermined threshold value is acquired during a period when vehicle speed of a vehicle is greater than or equal to a vehicle speed threshold value, as a counted number of times of end-abutting and retain a number of times that the end-abutting steering angle having variation less than or equal to the predetermined threshold value is acquired during a period when the vehicle speed is less than a vehicle speed threshold value, as a provisional counted number of times and, when the vehicle speed changes from less than the vehicle speed threshold value to greater than or equal to the vehicle speed threshold value, assign a value of the provisional counted number of times to the counted number of times of end-abutting. When the counted number of times of end-abutting is greater than or equal to the first threshold number of times, the relearning unit may reset the learned terminal position to an initial value.

[0303] There are some cases where in the stop state, there is a possibility that a tire is in contact with a curb or the like and an end-abutting steering angle is mistakenly acquired before end-abutting actually occurs. Because of this configuration, deferring relearning until the vehicle transitions from the stop state to the travel state enables incorrect learning to be prevented.

[0304] (7) The end-abutting detection unit may, when no end-abutting occurs in a case of steering to one of the left side and the right side and end-abutting occurs in a case of steering to the other of the left side and the right side, calculate a rack stroke, based on the learned terminal position learned on the one of the left side and the right side and the end-abutting steering angle acquired on the other of the left side and the right side. When error of the calculated rack stroke does not fall within a predetermined allowable range, the end-abutting detection unit does not use the end-abutting

steering angle acquired on the other of the left side and the right side for calculation of variation in the end-abutting steering angles.

[0305] When error of the rack stroke calculated based on the end-abutting steering angles does not fall within a predetermined allowable range, there is a possibility of erroneous detection of an end-abutting steering angle. Not using such an end-abutting steering angle enables incorrect learning to be prevented.

[0306] (8) The end-abutting detection unit may, when end-abutting occurs both in a case of steering to the left side and in a case of steering to the right side, calculate rack stroke, based on the end-abutting steering angles acquired in the case of steering to the left side and in the case of steering to the right side, and when the calculated rack stroke is less than a predetermined lower limit, assign a value of the counted number of times of end-abutting to the provisional counted number of times and resets a value of the counted number of times of end-abutting to 0.

[0307] When end-abutting occurs both in a case of steering to the left side and in a case of steering to the right side and the rack stroke calculated based on the end-abutting steering angles is less than a predetermined lower limit as described above, there is a possibility of erroneous detection of an end-abutting steering angle. Thus, moving a value of the counted number of times of end-abutting to the provisional counted number of times and temporarily deferring relearning enable incorrect learning to be prevented.

REFERENCE SIGNS LIST

[0308]	1 Steering wheel
[0309]	2i Column input shaft
[0310]	2o Column output shaft
[0311]	3 Reduction gear
[0312]	4 Intermediate shaft
[0313]	4a, 4b Universal joint
[0314]	4c Shaft member
[0315]	5 Pinion rack mechanism
[0316]	5a Pinion
[0317]	5b Rack
[0318]	6a, 6b Tie rod
[0319]	7a, 7b Hub unit
[0320]	8L, 8R Steered wheel
[0321]	10 Torque sensor
[0322]	11 Ignition key
[0323]	12 Vehicle speed sensor
[0324]	13 Battery
[0325]	14 Steering angle sensor
[0326]	20 Motor
[0327]	30 Controller
[0328]	40 Basic command value calculation unit
[0329]	41, 64 Adder
[0330]	42, 82 Subtractor
[0331]	43 Current control unit
[0332]	44 PWM control unit
[0333]	45 Inverter
[0334]	46 Terminal position learning unit
[0335]	47 Control rotational displacement setting unit
[0336]	48 Differentiating unit
[0337]	49 Impact mitigation control unit
[0338]	50 Current detector
[0339]	51 Learning state determination unit
[0340]	52 Impact mitigation control output limiting unit
[0341]	53 Relearning determination unit

- [0342] 60 Spring constant table
- [0343] 61, 63 Multiplier
- [0344] 62 Viscosity constant table
- [0345] 65 Inverter
- [0346] 66, 79 Limiter
- [0347] 70 Output shaft torque calculation unit
- [0348] 71 Selection unit
- [0349] 72 First storage unit
- [0350] 73, 77 Delay unit
- [0351] 74 Rate limiter
- [0352] 75 Corrected position calculation unit
- [0353] 76 Second storage unit
- [0354] 78 Third storage unit
- [0355] 80 Stroke calculation unit
- [0356] 81 Offset error calculation unit
- [0357] 83 Terminal position correction unit
- [0358] 90 End-abutting detection unit
- [0359] 91 End-abutting steering angle range determination unit
- [0360] 92 Counting unit
- [0361] 93 Relearning unit
- [0362] 94 End-abutting steering angle storage unit
- [0363] 95 Vehicle state determination unit
- [0364] 96bL, 96bR, 96oL, 96oR End-abutting counter
- [0365] 97bL, 97bR, 97oL, 97oR Provisional end-abutting counter
- [0366] 98bL, 98bR, 98oL, 98oR On-center counter

1. A turning control device comprising:

a position detection unit configured to detect a steered position of a turning mechanism of a vehicle; a terminal position learning unit configured to learn a terminal position of the turning mechanism, based on the steered position detected by the position detection unit; and a relearning determination unit configured to determine necessity of relearning of the terminal position,

wherein the relearning determination unit includes:

an end-abutting detection unit configured to detect occurrence of end-abutting, the end-abutting being a state in which the turning mechanism is turned to the terminal position, and acquire an end-abutting steering angle, the end-abutting steering angle being a steering angle when occurrence of end-abutting is detected;

an end-abutting steering angle range determination unit configured to determine whether or not variation in the end-abutting steering angles acquired multiple times is less than or equal to a predetermined threshold value; and

a relearning unit configured to, when the variation is less than or equal to the predetermined threshold value, reset the learned terminal position to an initial value.

2. The turning control device according to claim 1, wherein when occurrence of end-abutting is detected both in a case of steering to a left side and in a case of steering to a right side and each of variation in the end-abutting steering angles in the case of steering to the left side and variation in the end-abutting steering angles in the case of steering to the right side is less than or equal to the predetermined threshold value, the relearning unit calculates estimated rack stroke, based on the end-abutting steering angles and resets the initial value, based on the estimated rack stroke.

3. The turning control device according to claim 1 comprising

a counting unit configured to count a number of times of acquisition of the end-abutting steering angles having variation less than or equal to the predetermined threshold value,

wherein when the number of times of acquisition is greater than or equal to a first threshold number of times, the relearning unit resets the learned terminal position to the initial value.

4. The turning control device according to claim 3, wherein

the steering angle range determination unit stores the acquired end-abutting steering angle and determines whether or not variation in the end-abutting steering angles is less than or equal to the predetermined threshold value, based on a difference between the stored end-abutting steering angle and a steering angle when occurrence of end-abutting is detected, and

the counting unit resets counting of the number of times of acquisition when variation in the end-abutting steering angles is determined not to be less than or equal to the predetermined threshold value.

5. The turning control device according to claim 4, wherein

the counting unit

counts a number of times that it is determined that a second steering angle, the second steering angle being a steering angle when occurrence of end-abutting is detected, is closer to a neutral position of the turning mechanism than a first steering angle, the first steering angle being a steering angle located farthest from the neutral position among the stored end-abutting steering angles, and a difference between the first steering angle and the second steering angle is not less than or equal to the predetermined threshold value, as a number of times of being on-center, and

when the number of times of being on-center is greater than or equal to a second threshold number of times, resets counting of the number of times of acquisition.

6. The turning control device according to claim 3, wherein

the counting unit

includes a count storage unit configured to retain a number of times that the end-abutting steering angle having variation less than or equal to the predetermined threshold value is acquired during a period when vehicle speed of a vehicle is greater than or equal to a vehicle speed threshold value, as a counted number of times of end-abutting and retain a number of times that the end-abutting steering angle having variation less than or equal to the predetermined threshold value is acquired during a period when the vehicle speed is less than a vehicle speed threshold value, as a provisional counted number of times, and

when the vehicle speed changes from less than the vehicle speed threshold value to greater than or equal to the vehicle speed threshold value, assigns a value of the provisional counted number of times to the counted number of times of end-abutting, and

when the counted number of times of end-abutting is greater than or equal to the first threshold number of times, the relearning unit resets the learned terminal position to an initial value.

7. The turning control device according to claim 1, wherein

the end-abutting detection unit
when no end-abutting occurs in a case of steering to one of the left side and the right side and end-abutting occurs in a case of steering to the other of the left side and the right side, calculates a rack stroke, based on the learned terminal position learned on the one of the left side and the right side and the end-abutting steering angle acquired on the other of the left side and the right side, and
when error of the calculated rack stroke does not fall within a predetermined allowable range, does not use the end-abutting steering angle acquired on the other of the left side and the right side for calculation of variation in the end-abutting steering angles.

8. The turning control device according to claim 6, wherein
the end-abutting detection unit, when end-abutting occurs both in a case of steering to the left side and in a case of steering to the right side, calculates rack stroke, based on the end-abutting steering angles acquired in the case of steering to the left side and in the case of steering to the right side, and
when the calculated rack stroke is less than a predetermined lower limit, assigns a value of the counted

number of times of end-abutting to the provisional counted number of times and resets a value of the counted number of times of end-abutting to 0.

9. The turning control device according to claim 1 comprising:
a command value calculation unit configured to calculate, based on an operation acting on a steering operation unit of a vehicle, a current command value for an actuator providing the turning mechanism with steering assist force;
a command value correction unit configured to, when a steered position detected by the position detection unit is in a vicinity of the terminal position learned by the terminal position learning unit, correct the current command value calculated by the command value calculation unit; and
a driving unit configured to drive-control the actuator, based on the current command value corrected by the command value correction unit.

10. A turning device comprising:
the turning control device according to claim 9; and
an actuator configured to, drive-controlled by the turning control device, turn a steered wheel of the vehicle.

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