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Heating, air conditioning, and dehumidification (“HACD”) systems based on legacy form factor

Abstract

A heating, air conditioning, and dehumidifier (“HACD”) system and methods of use are provided. The system may include an outdoor system and an indoor system. The outdoor system may include a compressor, a fully controlled condenser fan, a condenser coil, and a 3-way reheat valve. The indoor system may include a hot gas reheat coil, a thermal expansion valve (“TXV”), a check valve, a metering device, a blower motor, and an evaporator. The HACD system may include a refrigerant and a refrigerant piping system. The refrigerant piping system may include a first pipe being a compressor discharge pipe, a second pipe, a third pipe, and a fourth pipe being a suction line pipe. The indoor system may be configured to be confined to a form factor footprint such that physical dimensions of the indoor system are less than or equal to about 28×25.5×52 inches.

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Background/Summary

FIELD OF TECHNOLOGY

(1) Aspects of the disclosure relate to heating, air conditioning, and dehumidification ("HACD") systems.

BACKGROUND OF THE DISCLOSURE

(2) Typical HACD systems have compressor units located only indoors. These typical HACD configurations may have several drawbacks.

(3) First, compressor units are usually located separately as indoor units. These indoor compressor units therefore occupy much-needed indoor space.

(4) Second, compressor units are loud. Compressor units located indoors can keep people awake and cause undue stress because of noise volume.

(5) Third, compressor units are not efficient. Because efficiency of the indoor compressor units is poor, indoor compressor units cause excessive energy consumption. Excess energy consumption can be costly.

(6) Fourth, compressor units are not easily installable. HACD system installers are accustomed to installing standardized equipment designed for standardized spaces. To require HACD professionals to install an entirely separate unit is challenging for both HACD installers and customers who must pay for these services.

(7) Therefore, it would be desirable to include the indoor compressor unit in an outdoor unit.

(8) It would also be desirable to achieve a smaller system footprint while maintaining a standard form factor. HACD system engineers and contractors do not typically deviate from practical installation practices. Therefore, it would be desirable to provide a system with a form factor and footprint size that engineers and contractors typically use.

(9) It would be even further desirable to provide and efficient HACD systems with a dehumidification mode, within a confined form factor footprint, that will continue dehumidifying while space cooling is already satisfied.

SUMMARY OF THE DISCLOSURE

(10) FIG. **1** shows how an AC and heat pump system work in general. A typical AC system may include an evaporator **102** (EV). A typical AC system may include compressor **104**. A typical AC system may include metering device **106**. A typical AC system may include condenser **108** (COND).

(11) Evaporator **102** may transport, via, e.g., "Lo" pressure, refrigerant to compressor **104**. Compressor **104** may transport, via, e.g., "Hi" pressure, refrigerant to condenser **108**. Condenser **108** may transport, via, e.g., "Hi" pressure, refrigerant to metering device **106**. Metering device **106** may transport, via, e.g., "Lo" pressure, refrigerant to evaporator **102**.

(12) A direction of flow of the refrigerant through a typical AC system may be, e.g., as follows: (1) from evaporator **102** to compressor **104**; (2) from compressor **104** to condenser **108**; (3) from condenser **108** to metering device **106**; and (4) from metering device **106** to evaporator **102**.

(13) A typical heat pump system may include indoor coil **112** (EV known as ("k.a.") evaporator). A typical heat pump system may include compressor **110**. A typical heat pump system may include metering device **114**. A typical heat pump system may include 4-way valve **116**. A typical heat pump system may include outdoor coil (COND k.a. condenser) **118**.

(14) Indoor coil (EV) **112** acting as a condenser may transport, via, e.g., "Hi" pressure, refrigerant

to metering device **114**. Metering device **114** may transport, via, e.g., “Lo” pressure, refrigerant to outdoor coil acting as evaporator (COND k.a. condenser) **118**. Outdoor coil acting as evaporator (COND k.a. condenser) **118** may transport, via, e.g., “Lo” pressure, refrigerant to compressor **110**, coupled via 4-way valve **116**. Compressor **110** may transport, via, e.g., “Hi” pressure, refrigerant to Indoor coil **112** (EV k.a. evaporator), coupled via 4-way valve **116**.

(15) A direction of flow of the refrigerant through a typical heat pump system may be, e.g., as follows: (1) from indoor coil **112** (EV k.a. evaporator) to metering device **114**; (2) from metering device **114** to outdoor coil acting as evaporator (COND k.a. condenser) **118**; (3) from outdoor coil acting as evaporator (COND k.a. condenser) **118** to compressor **110**, via 4-way valve **116**; and (4) from compressor **110** to indoor coil (EV) **112** acting as a condenser, via 4-way valve **116**.

(16) HACD systems and methods of use are provided. HACD systems and methods incorporating a dehumidifier with an A/C and heating unit are provided.

(17) The HACD system and methods may include three modes: (1) an AC mode; (2) a heating mode; and (3) a dehumidification mode. The three modes may provide three refrigerant flow sequences. The AC mode may provide cool air in a first refrigerant flow sequence. The heating mode may provide warm air in a second refrigerant flow sequence. The dehumidification mode may provide natural dry air in a third refrigerant flow sequence.

(18) In the AC mode, the HACD system may cool down space air (e.g., ambient air and/or air in the vicinity of the HACD system). Space air may enter the HACD system. Space air may be cooled via air being circulated over (EV) cooling coil included in the HACD system. The cooling coils may include a refrigerant. The refrigerant included in the (EV) cooling coil may be reduced to a lower pressure to cool down the space air. As the refrigerant is reduced to a lower pressure, moisture from the space air may condense over the cooling coil and thereby may be removed from the space air.

(19) In the dehumidification mode, adding to the embodiment as set forth herein, the HACD system may dehumidify the ambient air without substantially increasing and/or decreasing the temperature of the air. In the dehumidification mode, ambient air may enter the HACD system. Moisture may be condensed out of the ambient air using the cooling coils. As the ambient air is cooled, the temperature of the ambient air may drop. The HACD system may reheat the cooled ambient air using reheat coils. The reheat coils may reheat the temperature of the cooled ambient air back to its original temperature.

(20) The HACD systems and methods may include an HACD system with a confined footprint. The HACD systems and methods of use may be installed and employed by agents, contractors, and engineers that handle typical heating, air conditioning (“AC”), and/or dehumidification systems. The HACD systems and methods of use may be configured for enhanced energy efficiency. The HACD systems and methods of use may be configured for indoor and/or outdoor noise reduction.

(21) The HACD systems and methods may include an outdoor system. The outdoor system may include a compressor. The outdoor system may include a condenser. The outdoor system may include a fully controlled condenser fan. The outdoor system may include a condenser coil. The outdoor system may include a 3-way reheat valve, controls, etc.

(22) The HACD systems and methods may include an indoor system. The indoor system may include an evaporator. The indoor system may include a hot gas reheat coil. The indoor system may include a thermal expansion valve (“TXV”). The TXV may control an amount of refrigerant released into the evaporator. The TXV may regulate a temperature of refrigerant that flows out of the evaporator.

(23) The indoor system may include a check valve. The indoor system may include a metering device. The indoor system may include a blower motor.

(24) The HACD systems and methods may include a refrigerant piping system. The refrigerant piping system may be configured to transport refrigerant throughout the HACD system.

(25) The refrigerant piping system may include a first pipe. The first pipe may be a compressor

discharge pipe. The first pipe may be coupled from the compressor to the 3-way reheat valve. The 3-way reheat valve may be configured to output refrigerant to the hot gas reheat coil.
(26) The refrigerant piping system may include a second pipe. The second pipe may be coupled, via the check valve, from the hot gas reheat coil to the condenser coil.
(27) The refrigerant piping system may include a third pipe. The third pipe may be coupled, via the metering device, from an outlet of the condenser coil to an input of the evaporator.
(28) The refrigerant piping system may include a fourth pipe. The fourth pipe may be a suction line pipe. The fourth pipe may be coupled from an outlet of the evaporator to an input of the compressor. The fourth pipe may be configured to output the refrigerant to the compressor.
(29) In some embodiments, the indoor system may be configured to a confined form factor footprint. For example, the physical dimensions of the indoor system may be less than or equal to about 60×30×30 inches.

Description

BRIEF DESCRIPTION OF THE DRAWINGS
(1) The objects and advantages of the invention may be apparent upon consideration of the following detailed description, taken in conjunction with the accompanying drawings, in which like reference characters refer to like parts throughout.
(2) FIG. **1** shows an illustrative diagram of an AC and heat pump system.
(3) FIG. **2**A shows an illustrative diagram of a HACD system in accordance with principles of the disclosure.
(4) FIG. **2**B shows an illustrative diagram of a HACD system in accordance with principles of the disclosure.
(5) FIG. **3** shows an illustrative diagram of an outdoor section of the HACD system in accordance with the disclosure.
(6) FIG. **4** shows another illustrative diagram of an outdoor HACD system in accordance with the disclosure.
DETAILED DESCRIPTION OF THE DISCLOSURE
(7) HACD systems and methods of use are provided.
(8) HACD systems and methods of use may include 3 different operational modes. The 3 different operational modes may include: (1) cooling (AC); (2) heating (same as AC with reverse cycle integral valves that move a refrigerant in a different direction); and (3) dehumidification.
(9) HACD systems and methods may include dehumidification control for humidity and AC control for temperature. Normally, cooling control is a priority. HACD systems and methods may include dehumidification control that is not a priority.
(10) HACD systems and methods may standardize a split dehumidification system (with an outdoor condenser and an indoor air handler with reheat coil capabilities).
(11) The HACD systems and methods may include a thermostat main bar. The thermostat main bar may include a cooling mode option, a heating mode, and a dehumidification option. The HACD systems and methods may include all three modes in the same indoor unit without losing any efficiency.
(12) The HACD systems and methods may be configured to use the compressor while switching between the modes (heating, cooling, and dehumidification). The HACD systems may be configured to not use the compressor for mode changes, thereby avoiding compressor delays.
(13) An AC system normally dehumidifies while simultaneously cooling because the AC removes moisture from the air. Thus, HACD systems and methods may include dehumidification without cooling. And in some embodiments, HACD systems and methods may be used at room temperature in active dehumidification mode.

(14) In some embodiments, HACD systems and methods may remove excess heat. While in the dehumidification mode, the condenser may accumulate enough cooling to overcome compressor heat loss. The condenser may gain cooler temperature supply air while in dehumidification mode. The HACD systems and methods may not lose any heat while running cooling, heating, and dehumidification modes.
(15) The HACD systems and methods may achieve full capacity with a third of the space compared to typical AC and dehumidification systems with similar British Thermal Unit ("BTU") capacity (e.g., the footprint or volume of the equipment may be about one third that of typical systems with similar capacity). The HACD systems and methods may include a form factor footprint of physical dimensions not greater than, e.g., 28″×25.25″×52″.
(16) HACD systems and methods may include coils. The coils may include pipes, e.g., copper pipes and aluminum pipes. The pipes may include fins.
(17) HACD systems and methods may include a hot gas reheat coil. HACD systems and methods may include a cooling coil. The cooling coil may cause cooling by dehumidification (e.g., condensation from the air). HACD systems and methods may include the hot gas reheat coil to offset the cooling caused by the cooling coil, which is caused by the dehumidification (e.g., the condensation from the air).
(18) The HACD systems and methods may include a hot gas reheat line, for example, the hot gas reheat coil, pipe, and valve. A typical AC and dehumidification system may include a compressor, a 4-way valve, an accumulator, a TXV, and loop for refrigerant or liquid to circulate, however may not have a hot gas reheat line. The hot gas reheat line may include refrigerant that arrives hot. The hot gas reheat line may include refrigerant that outputs hot gas. The hot gas may be hot and/or slightly warm.
(19) The gas reheat line may be insulated. The gas reheat line may be insulated to maintain a temperature. The HACD systems and methods may handle outdoor ambient air. The hot gas reheat coil may heat the outdoor ambient air. The hot gas reheat coil may blow the heated outdoor ambient air back into the conditioned space.
(20) HACD systems and methods may bring hot gas from the outside unit to the inside. Cooled air from the HACD system may be tempered with heat from the hot gas reheat coil of the HACD system. HACD systems and methods may then produce neutral-temperature air (e.g., room temperature or 20±5° C.) to dehumidify space air.
(21) The HACD systems and methods may include a condenser. HACD systems and methods may include a fully controlled condenser fan. HACD systems and methods may include a condenser coil. The fully controlled condenser fan may be configured to transport air over the condenser coil.
(22) The HACD systems and methods may include two condenser coils. The HACD systems and methods may include a first condenser coil. The first condenser coil may be cold. The HACD systems and methods may include a second condenser coil. The second condenser coil may be hot.
(23) HACD systems and methods may include an evaporator. The evaporator (also referred to as an evaporator coil) may be cold to touch because it may extract heat from ambient air. The evaporator may become cold immediately because heat may pass through a wall of the evaporator pipe and into refrigerant inside the evaporator pipe. The condenser may be hot to touch because the condenser may extract cold from the air, thereby cooling refrigerant within the condenser pipe. The evaporator coil and indoor condenser coil may be located within the same cabinet in the indoor system. The condenser may be located on top of the HACD systems. The evaporator may be located on the bottom of the HACD systems.
(24) The HACD systems and methods may include a compressor. The compressor may be a pump that circulates a chemical refrigerant. The compressor may pump heated refrigerant from the evaporator coil to the condenser coil. The compressor may be located outdoors. The compressor may include a condenser coil. The condenser coil may be configured to operate in dehumidification mode. The condenser coil may not use electric heat.

(25) HACD systems and methods may include a hot gas reheat coil that is configured to become the first condenser coil. For efficiency, HACD systems and methods may not expel heat outside. HACD systems and methods may therefore use heat within the same system to conserve energy.
(26) HACD systems and methods may include using both the condenser coil and the hot gas reheat coil simultaneously to achieve the desired form factor footprint. By contrast, typical AC and dehumidification systems will not use the outdoor condenser coil while dehumidifying. The simultaneous use of both the condenser coil and the hot gas reheat coil in the HACD systems and methods may reduce energy consumption.
(27) HACD systems and methods may modulate automatically between the hot gas reheat coil and the condenser coils. HACD systems and methods may not require modulation between the hot gas reheat coil and the outdoor condenser coil. This may lead to less servicing, maintenance, and mechanics required for the system.
(28) HACD systems and methods may include a heat pump. HACD systems and methods may include a variable-speed compressor. HACD systems and methods may include a 3-way heat reclaim valve. The 3-way heat reclaim valve may avoid compressor interruption while switching between cooling, heating, and/or dehumidification modes. HACD systems and methods may be controllable with, e.g., remote controls and digital applications.
(29) HACD systems and methods may include a metering device. The metering device may include an orifice and/or other types of devices used in refrigeration and air conditioning systems. The orifice may restrict a flow of the refrigerant within the system. The orifice may have a diameter between any suitable range, e.g., about 0.1 mm to about 10 mm. The metering device may restrict the flow of hot refrigerant to the condenser.
(30) The compressor may create high pressure at the metering device. Pressure may be higher within the coils between the metering device, condenser coils and the
(31) compressor. Pressure may be lower within the coils between the metering device, the evaporator coil, and the compressor.
(32) HACD systems and methods may include an air handler. The air handler may include the evaporator.
(33) In some embodiments, the evaporator coil may be configured to become the condenser. In some embodiments the condenser may be configured to become the evaporator coil.
(34) HACD systems and methods may include a 3-way valve. The 3-way valve may be a hot gas reheat valve. The 3-way valve may feed the refrigerant through a heat coil to the outdoor condenser.
(35) In some embodiments, HACD systems and methods may include a 4-way valve. The 4-way valve may reverse a flow of refrigerant in the HACD system.
(36) HACD systems and methods may include a suction line. The suction line may include a pipe with an inner width between about ⅜″ to ⅞″ in diameter.
(37) HACD systems and methods may include a 1-way valve. The 1-way valve may be a check valve. The 1-way valve may direct a flow of refrigerant from the compressor to the inlet of the hot gas reheat coil. The hot gas reheat coil may release the heat from itself to the surroundings. The check valve may control the flow of refrigerant from the hot gas reheat coil to the condenser coil. The condenser coil may be used for additional heat rejection of the refrigerant from the hot gas reheat coil.
(38) HACD systems and methods may include a blower motor. The blower motor may be configured to transport air over the evaporator.
(39) HACD systems and methods may include a TXV. The TXV may be configured to control an amount of refrigerant released into the evaporator.
(40) HACD systems and methods may include control feedback circuitry. HACD systems and methods may include humidity control from 0-100% humidity. 40-60% humidity may be a normal range of humidity indoors. 50% humidity may be an optimal humidity level for human comfort.

(41) The HACD systems and methods may include an accumulator. The accumulator may act as a receiver during heating and defrost cycles when system imbalance or an overcharge from field service could result in excessive liquid refrigerant in the system. The accumulator may store the refrigerant until needed and feed it back to the compressor at an acceptable rate.
(42) The HACD systems and methods for indoor sections may occupy varying physical dimensions and form factor footprints. For example, the HACD systems and methods may include forms with capacities up to 2 tons, 3 tons, 4 tons, and 5 tons. The HACD systems up to 2 tons may occupy a form factor footprint of 20″×20″×40″. The HACD systems up to 3 tons may occupy a form factor footprint of 23″×20″×42″. The HACD systems up to 4 tons may occupy a form factor footprint of 28″×21.25″×48″. The HACD systems up to 5 tons may occupy a form factor footprint of 28″×25.25″×52″. The HACD systems and methods may preferably occupy a form factor footprint of not greater than 28″×25.25″×52″.
(43) The HACD systems may not be custom built. The HACD systems may be produced in standardized forms. The HACD systems may be standardized to standard unit sizes and form factor footprints. The HACD systems may not include a modulator valve. Modulator valves may be commonly used in AC and dehumidification systems.
(44) The HACD systems and methods may include an indoor system. The HACD systems and methods may include an outdoor system. The HACD systems and methods may include a compressor. The compressor may be deployed in the outdoor system to avoid losing space indoors. An system with an indoor compressor may be confined inside a residential, commercial, or industrial site to a standardized form factor, footprint, and size. Further, using a compressor outdoors has an added advantage of avoiding compressor noise caused by indoor use.
(45) The indoor and outdoor systems may include a refrigerant. The indoor and outdoor systems may include a refrigerant piping system. The refrigerant piping system may be deployed to transport the refrigerant throughout the indoor and outdoor systems.
(46) The outdoor system may include an outdoor condenser. The outdoor system may include the fully controlled compressor. The outdoor system may include the fully controlled condenser fan. The condenser may include the first condenser coil. The outdoor system may include the 3-way reheat valve. The outdoor system may include the 4-way valve.
(47) The indoor system may include an evaporator. The indoor system may include an evaporator coil.
(48) The indoor system may include a hot gas reheat coil. The indoor system may include a hot gas reheat valve. The indoor system may include an indoor condenser. The indoor condenser may include a second condenser coil. The hot gas reheat valve may activate an indoor condenser. Refrigerant, passing through the hot gas reheat coil, may be heated by the hot gas reheat coil. The refrigerant may be heated up to about, e.g., 120° C.
(49) The indoor system may include a TXV. The indoor system may include a check valve. The indoor system may include a metering device. The indoor system may include a blower motor.
(50) The hot gas reheat coil may be located above a blower motor. The hot gas reheat coil may be located a threshold distance away from the evaporator, thereby avoiding heat loss that would occur if the hot gas reheat coil and evaporator coil were adjacent or close to each other.
(51) The condenser may include four pipes. The four pipes may include two pipes for conventional cooling and or heat pump. The four pipes may include two pipes for heat extraction. The four pipes may be included in the refrigerant piping system.
(52) The refrigerant piping system, and method of use thereof, may include a first pipe. The refrigerant may move through the first pipe. The first pipe may be a compressor discharge pipe. The first pipe may be coupled from the compressor to the 3-way reheat valve. The 3-way reheat valve may be configured to output the refrigerant to the outdoor condenser coil and the hot gas reheat coil. In some embodiments, the 3-way reheat valve may be configured to output the refrigerant to the condenser coil.

(53) The refrigerant piping system, and method of use thereof, may include a second pipe. The refrigerant may move through the second pipe. The second pipe may be coupled, via the check valve, from the hot gas reheat coil to the outdoor condenser coil. The check valve may be configured to output the refrigerant to the condenser coil.
(54) The refrigerant piping system, and method of use thereof, may include a third pipe. The third pipe may be coupled, via the metering device, from an outlet of the condenser coil to an input of the evaporator. The metering device may be configured to output the refrigerant to the evaporator.
(55) The refrigerant piping system, and method of use thereof, may include a fourth pipe. The fourth pipe may be a suction line pipe. The fourth pipe may be coupled from an outlet of the evaporator to an input of the compressor. The fourth pipe may be configured to output the refrigerant to the compressor.
(56) The HACD systems and methods may include a loop. The loop may be used for circulating pool water. The loop may carry a refrigerant or liquid to boiler inlet and or a reheat coil, to reuse the energy of the heat where needed.
(57) The indoor system may be configured to be confined to a form factor footprint. For example, the physical dimensions of the indoor system may be less than or equal to about 30×30×60 inches.
(58) Systems and methods described herein are illustrative. Systems and methods in accordance with this disclosure may now be described in connection with the figures, which form a part hereof. The figures show illustrative features of systems and method steps in accordance with the principles of this disclosure. It is to be understood that other embodiments may be utilized, and that structural, functional and procedural modifications may be made without departing from the scope and spirit of the present disclosure.
(59) The steps of methods may be performed in an order other than the order shown or described herein. Embodiments may omit steps shown or described in connection with illustrative methods. Embodiments may include steps that are neither shown nor described in connection with illustrative methods.
(60) Illustrative method steps may be combined. For example, an illustrative method may include steps shown in connection with another illustrative method.
(61) Systems may omit features shown or described in connection with illustrative systems. Embodiments may include features that are neither shown nor described in connection with the illustrative systems. Features of illustrative systems may be combined. For example, an illustrative embodiment may include features shown in connection with another illustrative embodiment.
(62) FIG. **2**A shows an illustrative diagram of HACD system **200**.
(63) FIG. **2**B shows an illustrative diagram of HACD system **200**. HACD system **200** is identical to HACD system **200** in FIG. **2**A. However, HACD system **200** shown in FIG. **2**B shows HACD system **200** in a horizontal position.
(64) HACD system **200** may include loop **201**. Loop **201** may include an AC/heating loop. HACD system **200** may include loop **203**. AC/heating may be provided through third pipe (3) **126** and fourth pipe (4) **124**, included in loop **201**. Loop **203** may include a dehumidification loop. Dehumidification may be provided via first pipe (1) **210** and second pipe (2) **208**, included in loop **203**.
(65) HACD system **200** may include air handler cabinet **202**. Air handler cabinet **202** may include evaporator **204**. Air handler cabinet **202** may be a cabinet used to hold devices that regulate and circulate air in HACD system **200**. Evaporator **204** may be a heat exchanger device that transfers heat from its surroundings (e.g., ambient air) into a refrigerant causing a phase change (i.e., evaporation). The refrigerant may be a fluid used to undergo repeated phase transitions between a liquid and a gas.
(66) Air handler cabinet **202** may include hot gas reheat coil **206**. Hot gas reheat coil **206** may be used during an HACD cooling cycle to dehumidify cool air coming from evaporator **204**. A portion of refrigerant going to condenser **214** in HACD system **200** may be rerouted to the hot gas reheat

coil **206**. The refrigerant may be reheated at the hot gas reheat coil **206**. Hot gas reheat coil **206** may then heat ambient air via the reheated refrigerant. In some embodiments, most or all refrigerant may be sent to hot gas reheat coil **206** for reheating. In some embodiments, hot gas reheat coil **206** may use high pressure, high temperature compressor discharge gas to reheat air leaving evaporator **204**.

(67) Air handler cabinet **202** may include blower motor **232**. Blower motor may help transport air throughout HACD system **200**. Hot gas reheat coil **206** may emit air and refrigerant to the outside of air handler cabinet **202**.

(68) HACD system **200** may include condenser **214**. Condenser **214** may be a heat exchanger device used to condense a gas into a liquid through cooling. Condenser **214** may include condenser coil **216**. Condenser coil **216** may release heat that was absorbed by evaporator **204**.

(69) Condenser **214** may include compressor **218** (C). Compressor **218** may pressurize refrigerant in the HACD system, thereby turning incoming refrigerant into hot, high-pressured gas. Compressor **218** may push the refrigerant through a refrigerant piping system included in HACD system **200**. The refrigerant piping system may include four pipes. The four pipes may include two pipes for conventional cooling (e.g., second pipe (2) **208** and third pipe (3) **226**) and two pipes for heat extraction (e.g., first pipe (1) **210** and fourth pipe (4) **224**).

(70) HACD system **200** may include 4-way valve **220**. 4-way valve **220** may be connected to compressor **218**. 4-way valve **220** may include a 4-way valve connection. The 4-way valve connection may be reversing valve **230**. 4-way valve **220** or reversing valve **230** may be used to reverse the heating/cooling cycle of HACD system **200**.

(71) Pressure level inside the pipes in the refrigerant piping system may be indicated by “Lo” or “Hi.” “Lo” pressure may indicate relatively lower pressure in a pipe. “Hi” pressure may indicate relatively higher pressure in a pipe. In some embodiments, “Lo” pressure may correspond to a relatively cooler temperature of refrigerant in a pipe. In some embodiments, “Hi” pressure may correspond to a relatively hotter temperature of refrigerant in a pipe.

(72) Compressor **218** may transport, via first pipe (1) **210**, hot refrigerant to hot gas reheat coil **206**. First pipe (1) **210** may be a compressor discharge pipe. The hot refrigerant may be about 120° C.

(73) First pipe (1) **210** may be coupled from compressor **218** to 3-way hot gas reheat valve **222**. 3-way hot gas reheat valve **222** may divert refrigerant from entering condenser coil **216** to hot gas reheat coil **206**, thereby reheating the circulating space air. 3-way hot gas reheat valve **222** may be configured to output the refrigerant to condenser coil **216** and/or hot gas reheat coil **206**.

(74) Compressor **218** may transport, via first pipe (1) **210**, refrigerant to hot gas reheat coil **206**, through 3-way hot gas reheat valve **222**. The pressure in the first pipe (1) **210** may be “Hi.”

(75) Hot gas reheat coil **206** may transport, via second pipe (2) **208** and an internal pipe inside condenser **214**, refrigerant to compressor **218**, through 3-way hot gas reheat valve **222**. Second pipe (2) **208** may be coupled via check valve **212** (CKV). Check valve **212** may be a one-way valve. Check valve **218** may allow flow through it in only one direction. Check valve **212** may be configured to output the refrigerant to compressor **218**.

(76) The pressure in second pipe (2) **208** may be almost as “Hi” as the pressure in first pipe (1) **210**. The temperature in second pipe (2) **208** may vary according to HACD system mode and environmental conditions. The pressure in the internal pipe inside condenser **214** transporting the refrigerant through hot gas reheat valve **222** to compressor **218** may be “Hi.”

(77) Condenser coil **216** may transport refrigerant from an outlet, via a liquid receiver (not illustrated) in a third pipe (3) **226**, to evaporator **204**. Third pipe (3) **226** may be coupled via metering device **228** (M|D). The metering device **228** may be a device that decreases the temperature and pressure of the refrigerant moving from condenser **214** before it enters evaporator **204**. The metering device **228** may output the refrigerant to evaporator **204**. The metering device **228** may alter the pressure inside third pipe (3) **226** from “Hi” to “Lo.”

(78) Evaporator **204** may transport refrigerant from an outlet, via fourth pipe (4) **224**, which may be

called a suction line pipe, to compressor **218**. Fourth pipe (4) **224** may be coupled with a suction line accumulator (not illustrated). The suction line pipe may be a refrigerant pipe that runs under low pressure to the compressor **218**.

(79) Fourth pipe (4) **224** may be coupled from an outlet of evaporator **204** to an input of compressor **218** coupled with a suction line accumulator (not illustrated). Fourth pipe (4) **224** may be configured to output the refrigerant to compressor **218**.

(80) A direction of flow of refrigerant through the refrigerant piping system may be, e.g., as follows: (1) the refrigerant may flow, via first pipe **210** (1) from an outlet of compressor **218** through 3-way hot gas reheat valve **222** into an input of hot gas reheat coil **206**; (2) the refrigerant may flow, via second pipe **208** (2) from an outlet of hot gas reheat coil **206** through check valve **212** into an input of the condenser coil **216**; (3) the refrigerant may flow, via third pipe **226** (3) from an outlet of condenser coil **216** through metering device **228** into an input of the evaporator **204**; and (4) the refrigerant may flow, via fourth pipe **224** (4) from an outlet of evaporator **204** into an input of compressor **218**.

(81) FIG. **3** shows an illustrative diagram of HACD system according to the embodiments. The HACD system may be installed outdoors.

(82) The HACD system may include outdoor system **302**. Outdoor system **302** may have one or more features in common with HACD system **200**. Outdoor system **302** may include fully controlled condenser fan **304**. Outdoor system **302** may include compressor **306**. Outdoor system **302** may include condenser coil **308**. Outdoor system **302** may include 3-way reheat valve (1) **322** (also referred to as a 3-way hot gas reheat valve).

(83) Compressor **306** may be coupled to fully controlled condenser fan **304** via 3-way reheat valve (1) **322**. Compressor **306** may be coupled to fully controlled condenser fan **304** via 4-way reversing valve (3) **318**. 4-way reversing valve (3) **318** may provide both heating and cooling (airflow going the opposite direction to heating).

(84) Outdoor system **302** may include a connection to an indoor system. Outdoor system **302** may include a refrigerant piping system. The refrigerant piping system may be connected to the indoor system. The refrigerant piping system may include first pipe **314**. The refrigerant piping system may include second pipe **316**. The refrigerant piping system may include third pipe **312**. The refrigerant piping system may include fourth pipe **310**.

(85) First pipe **314** may include an inner pipe diameter of about ½″. Second pipe **316** may include an inner pipe diameter of about ½″. Third pipe **312** may include an inner pipe diameter of about ⅜″. Fourth pipe **310** may include an inner pipe diameter of about ⅞″.

(86) First pipe **314** may have one or more features in common with first pipe (1) **110**. Second pipe **316** may have one or more features in common with second pipe (2) **108**. Third pipe **312** may have one or more features in common with third pipe (3) **126**. Fourth pipe **310** may have one or more features in common with fourth pipe (4) **124**.

(87) FIG. **4** shows an illustrative diagram of outdoor HACD system **402** according to the embodiments. Outdoor HACD system **402** may have one or more features in common with one or more of HACD system **200** and outdoor HACD system **302**.

(88) Outdoor HACD system **402** may include electric conduit **404**. Outdoor HACD system **402** may include a refrigerant piping system.

(89) The refrigerant piping system may include four pipes. The refrigerant piping system may include first pipe **410**. The refrigerant piping system may include second pipe **412**. The refrigerant piping system may include third pipe **408**. The refrigerant piping system may include fourth pipe **406**.

(90) The first pipe **410** may have one or more features in common with first pipe **314**. Second pipe **412** may have one or more features in common with second pipe **316**. Third pipe **408** may have one or more features in common with third pipe **312**. Fourth pipe **406** may have one or more features in common with fourth pipe **310**.

(91) All valves, controls, and accessories may be located in either an indoor unit and/or an outdoor unit.
(92) Thus, HACD systems and methods of use are provided. Persons skilled in the art may appreciate that the present invention can be practiced by other than the described embodiments, which are presented for purposes of illustration rather than of limitation. The present invention is limited only by the claims that follow.

Claims

1. A heating, air conditioning, and dehumidification (“HACD”) system comprising: an outdoor system, the outdoor system comprising: a compressor; a fully controlled condenser fan; a condenser coil; a 4-way reheat valve; and/or a 3-way reheat valve; an indoor system, the indoor system comprising: a hot gas reheat coil; a thermal expansion valve (“TXV”); a check valve; a metering device; a blower motor; and a flat evaporator coil; a refrigerant; and a refrigerant piping system, the refrigerant piping system comprising: a first pipe, the first pipe being a compressor discharge pipe, the first pipe being coupled from the compressor to the 3-way reheat valve, the 3-way reheat valve configured to output the refrigerant to the condenser coil and the hot gas reheat coil; a second pipe, the second pipe being coupled, via the check valve, from the hot gas reheat coil to the condenser coil, the check valve configured to output the refrigerant to the condenser coil; a third pipe, the third pipe being coupled, via the metering device, from an outlet of the condenser coil to an input of the flat evaporator coil, the metering device configured to output the refrigerant to the flat evaporator coil; and a fourth pipe, the fourth pipe being a suction line pipe, the fourth pipe being coupled from an outlet of the flat evaporator coil to an input of the compressor, the fourth pipe configured to output the refrigerant to the compressor; and wherein: the hot gas reheat coil is located a threshold distance from the flat evaporator coil thereby avoiding a heat loss; the fully controlled condenser fan is configured to transport air over the condenser coil; the blower motor is configured to transport air over the flat evaporator coil; the TXV is configured to control an amount of refrigerant released into the flat evaporator coil; and the indoor system is configured to be confined to a form factor footprint such that physical dimensions of the indoor system are less than or equal to about 28×25.5×52 inches.
2. The HACD system of claim 1 wherein the refrigerant piping system comprises two pipes for conventional cooling and two pipes for heat extraction.
3. The HACD system of claim 1 wherein the HACD system does not use electric heat.
4. The HACD system of claim 1 wherein a use of the refrigerant piping system, with the fully controlled condenser fan, causes the refrigerant to be transported first to the hot gas reheat coil and then to the condenser coil.
5. The HACD system of claim 1 wherein, in a dehumidification mode, the outdoor condenser rejects additional heat accumulated from compressor motor heat gain, thereby maintaining a net neutral temperature of supply air in the indoor system.
6. The HACD system of claim 1 wherein switching between modes does not require shutting off the compressor, thereby avoiding delays, the modes including heating, cooling, and dehumidification.
7. The HACD system of claim 1 wherein the compressor is a variable-speed compressor, and further wherein the variable-speed compressor and the 4-way valve are configured to switch between modes without requiring shutting off the compressor.
8. The HACD system of claim 1 wherein the hot gas reheat coil is located above the blower motor, and located a threshold distance from the flat evaporator coil, thereby avoiding heat loss.
9. The HACD system of claim 1 wherein the indoor system further comprises an accumulator, the accumulator storing the refrigerant until the refrigerant is needed by the system and sending the refrigerant to the compressor when the refrigerant is needed by the system.

10. The HACD system of claim 1 wherein the indoor system is configured to be confined to a form factor footprint such that physical dimensions of the indoor system are less than or equal to about 20×20×40 inches.

11. A method of using a heating, air conditioning, and dehumidification ("HACD") system, the method comprising using: an outdoor system, the outdoor system comprising: a compressor; a fully controlled condenser fan; a condenser coil; a 4-way reheat valve; and/or a 3-way reheat valve; an indoor system, the indoor system comprising: a hot gas reheat coil; a thermal expansion valve ("TXV"); a check valve; a metering device; a blower motor; and a flat evaporator coil; a refrigerant; and a refrigerant piping system, the refrigerant piping system comprising: a first pipe, the first pipe being a compressor discharge pipe, the first pipe being coupled from the compressor to the 3-way reheat valve, the 3-way reheat valve configured to output the refrigerant to the condenser coil and the hot gas reheat coil; a second pipe, the second pipe being coupled, via the check valve, from the hot gas reheat coil to the condenser coil, the check valve configured to output the refrigerant to the condenser coil; a third pipe, the third pipe being coupled, via the metering device, from an outlet of the condenser coil to an input of the flat evaporator coil, the metering device configured to output the refrigerant to the flat evaporator coil; and a fourth pipe, the fourth pipe being a suction line pipe, the fourth pipe being coupled from an outlet of the flat evaporator coil to an input of the compressor, the fourth pipe configured to output the refrigerant to the compressor; and wherein: the hot gas reheat coil is located a threshold distance from the flat evaporator coil thereby avoiding a heat loss; the fully controlled condenser fan is configured to transport air over the condenser coil; the blower motor is configured to transport air over the flat evaporator coil; the TXV is configured to control an amount of refrigerant released into the flat evaporator coil; and the indoor system is configured to be confined to a form factor footprint such that physical dimensions of the indoor system are less than or equal to about 28×25.5×52 inches.

12. The method of claim 11 wherein the refrigerant piping system comprises two pipes for conventional cooling and two pipes for heat extraction.

13. The method of claim 11 wherein the method does not use electric heat.

14. The method of claim 11 wherein the use of the refrigerant piping system, with the fully controlled condenser fan, causes the refrigerant to be transported first to the hot gas reheat coil and then to the condenser coil.

15. The method of claim 11 wherein, in a dehumidification mode, the condenser causes accumulation of a threshold level of temperature cooling to overcome a compressor heat loss, thereby cooling a temperature of supply air in the indoor system.

16. The method of claim 11 wherein switching between modes does not require use of the compressor, thereby avoiding delays, the modes including heating, cooling, and dehumidification.

17. The method of claim 11 wherein the compressor is a variable-speed compressor, and further wherein the variable-speed compressor and the 4-way valve are configured to switch between modes without requiring shutting off the compressor.

18. The method of claim 11 wherein the hot gas reheat coil is located above the blower motor, and located a threshold distance from the flat evaporator coil, thereby avoiding heat loss.

19. The method of claim 11 wherein the indoor system further comprises an accumulator, the accumulator storing the refrigerant until the refrigerant is needed by the system and sending the refrigerant to the compressor when the refrigerant is needed by the system.

20. The method of claim 11 wherein the indoor system is configured to be confined to a form factor footprint such that physical dimensions of the indoor system are less than or equal to about 20×20×40 inches.
