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### Extrusion and/or pultrusion device and method

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#### Abstract

A method and an extrusion- or pultrusion device for forming a profile product made from a plastically deformable material and/or viscoplastic material in a production direction, the device comprising, a rotating die, extending in a radial direction and a width direction, having two opposite first and second side walls and an outer circumferential surface extending in the width direction there between. The rotating die comprises a first side portion in connection to the first side wall and a second side portion in connection to the second side wall and a mid-portion extending between the first and second side portions, and a profile definition zone having a longitudinal direction coinciding with the production direction, a height direction and a width direction being perpendicular to the height direction, comprising a through channel comprising a first channel section followed by a second channel section downstream the first channel section with reference to the production direction. The rotating die is rotatable about an axis extending across the production direction and arranged to allow the outer circumferential surface to, while the rotating die rotates, exert a pressure onto a surface of the material when fed through the profile definition zone.

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## References Cited

### U.S. PATENT DOCUMENTS

Patent No.	Issued Date	Patentee Name	U.S. Cl.	CPC
2892212	12/1958	Rhodes	425/374	B29C 48/35
3394431	12/1967	Nalle, Jr.	425/327	B29C 48/08
3635631	12/1971	Fields	425/223	B29C 43/222
3782872	12/1973	Nalle, Jr.	425/382N	B29C 48/05
3869304	12/1974	Bogulslawski	118/410	B29C 48/305
4497619	12/1984	Popow	264/165	B29C 45/0441
4631161	12/1985	Popow	425/374	B29C 48/08
4820466	12/1988	Zachariades	264/294	B29C 48/09
4950151	12/1989	Zachariades	156/244.11	B29C 48/475
4963309	12/1989	Gohlish	425/374	B29C 48/07
5030079	12/1990	Benzing, II	425/374	B29C 48/06
5122049	12/1991	Baumgarten	425/188	B29C 48/256
5186960	12/1992	Walsh, Jr.	425/382.3	B29C 48/08
5453238	12/1994	Bardy	425/327	B29C 48/21
5904884	12/1998	Wakefield	425/363	B29C 48/07
6475420	12/2001	Numrich	264/280	B29C 48/35
2002/0011688	12/2001	Nodono	425/143	B29C 48/35
2004/0009255	12/2003	Lamoine	425/382.3	B29C 48/35
2006/0062868	12/2005	Pan	425/363	B29C 48/07
2006/0099289	12/2005	Fukumura	264/177.1	B29C 48/301
2010/0307663	12/2009	Kudo	156/397	B29D 30/60

2012/0018917	12/2011	Washiyama	425/72.1	B29C 33/58
2012/0171320	12/2011	Vargo	425/367	B29C 48/35
2012/0223451	12/2011	Hulseman	264/293	B29C 48/09
2013/0126543	12/2012	Bohrer	428/156	B29C 48/07
2013/0313732	12/2012	Hsu	425/174.4	B29C 48/0019
2013/0334729	12/2012	Denavit	264/172.14	B29C 43/46
2015/0104614	12/2014	Gallant	428/167	A44B 18/0049
2015/0360398	12/2014	Monnereau	425/141	B29C 48/12
2017/0165862	12/2016	Slama	N/A	B29C 48/35
2018/0193891	12/2017	Jansson Kragh	N/A	B29C 48/35
2018/0207698	12/2017	Jansson Kragh	N/A	B21C 35/023
2020/0368958	12/2019	Higashi	N/A	B29B 7/726
2022/0219367	12/2021	Jansson Kragh	N/A	B29C 70/52
2022/0242028	12/2021	Jansson Kragh	N/A	B29C 48/35
2022/0242029	12/2021	Jansson Kragh	N/A	B21C 25/02
2023/0382035	12/2022	Kragh	N/A	B29C 48/12
2023/0382036	12/2022	Kragh	N/A	B29C 48/911
2023/0405947	12/2022	Kragh	N/A	B21C 35/023

## FOREIGN PATENT DOCUMENTS

Patent No.	Application Date	Country	CPC
1735498	12/2005	CN	N/A
1543938	12/2004	EP	N/A
H0716643	12/1994	JP	N/A
504300	12/1995	SE	N/A
2051100	12/2019	SE	N/A
2051101	12/2019	SE	N/A
543400	12/2020	SE	N/A
543401	12/2020	SE	N/A
543402	12/2020	SE	N/A

## OTHER PUBLICATIONS

Patent Cooperation Treaty (PCT), International Search Report and Written Opinion for Application PCT/EP2021/078185 filed Oct. 12, 2021, mailed Jan. 11, 2022, International Searching Authority, EP, cited by applicant

“The future with 3D-iExtrusion Technology-Relieved Technologies,” 2020. pp. 1-4, XP055874042, URL: <http://web.archive.org/web/20200924003011/http://relieved.com/the-technology/>. cited by applicant

Sumerak et al., “Pultrusion Process Troubleshooting insulation.” XP055875402, Handbook of Troubleshooting Plastics Processes: A Practical Guide, 2012, New Jersey. cited by applicant

Subramanian, “Compression Molding,” XP055874115, Handbook of Troubleshooting Plastics Processes: A Practical Guide, 2012, New Jersey. cited by applicant

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## Background/Summary

## CROSS-REFERENCE TO RELATED APPLICATION

(1) This application is a 35 U.S.C. § 371 national stage application of PCT Application No. PCT/EP2021/078185 filed on Oct. 12, 2021 entitled “AN EXTRUSION AND/OR PULTRUSION DEVICE AND METHOD,” which claims priority to Sweden Patent Application No. 2051216-6 filed on Oct. 20, 2020, each of which are incorporated herein in their entirety by reference.

## TECHNICAL FIELD

(2) The invention relates to an extrusion and/or pultrusion device for forming a profile product made from a plastically deformable material in a production direction, said device comprising: a rotating die, extending in a radial direction and a width direction, having two opposite first and second side walls and an outer circumferential surface extending in the width direction there between, wherein the rotating die comprises a first side portion in connection to the first side wall and a second side portion in connection to the second side wall and a mid-portion extending between the first and second side portions, and a profile definition zone having a longitudinal direction coinciding with the production direction, a height direction and a width direction being perpendicular to the height direction, comprising a through channel comprising a first channel section followed by a second channel section downstream the first channel section with reference to the production direction, wherein the rotating die is rotatable about an axis extending across the production direction and arranged to allow the outer circumferential surface to, while the rotating die rotates, exert a pressure onto a surface of the material when fed through the profile definition zone the first channel section is circumferentially delimited by one or more walls and wherein the second channel section is circumferentially delimited by the circumferential surface of the rotating die and a channel portion comprising a counter-bearing opposite the rotating die and opposing first and second channel portion side walls between the rotating die and the counter-bearing.

(3) The invention also relates to a method for producing a profile product by use of such a device.

## BACKGROUND ART

(4) In the field of extrusion and/or pultrusion devices it is known using a rotating die immediately downstream a more traditional extrusion and/or pultrusion device using fixed or static walls. This type of extrusion with a rotating die is hereinafter referred to as 3D-extrusion and relates to that the rotating die operates in a pressurized zone in connection to the more traditional extrusion and/or pultrusion portion of the device, which differs 3D-extrusion from calendaring. The combination of the static walls in the first channel section and a rotating die in the second channel section gives the benefit of producing a profile product at a very high speed with maintained high quality of shape and imprint. It is thus an effective and relatively cheap production method that can be used for most materials that can be formed by use of extrusion, i.e. everything from e.g. plastic to aluminium.

(5) In PCT/SE2020/050451 it is discussed the advantage of optimising the first channel section and the second channel section with regard to minimum side leakage.

## SUMMARY OF THE INVENTION

(6) With reference to background art, there is however a need for an improved optimization along the profile definition zone, i.e. of the first channel section and the second channel section, when extruding materials that undergo plastic deformation. The load rate of the material in the first channel is controlled with reference to the load rate in the second channel by design of the respective first channel and second channel.

(7) The claimed device comprises both static and moving part and operates at high production rate and high pressure, and sometimes under high temperature and pressure. The settings for the device is dependent on material and type of profile to be produced. Experiments performed during a long time span has revealed that materials that undergo plastic deformation have special features/properties that have to be taken into account in order to produce a profile product with high quality in the claimed device and at the same time design the device in a way to make it withstand the high forces during operation. The following definitions are introduced in order to

give a reader information on suitable such materials and settings of the device.

#### (8) Plastically Deformable Materials

(9) Some materials, for example metals, deform permanently when subjected to a sufficiently high force. Permanent deformation is generally referred to as plastic deformation and materials which exhibit this type of deformation may therefore be denominated “plastically deformable materials”. The deformation behaviour in a plastic deformable material depends on the magnitude of the force that the material is subject to and is often described in so called “stress-strain curves”. Generally, plastically deformable material exhibit the following deformation behaviour.

(10) When the material is subjected to a small force it deforms elastically because the stress in the material increases the distance between the atoms in the material but does not affect their mutual arrangement. Therefore, the material reverts linearly to its original dimension when the force is removed. Thus, in this force region the material exhibits a linearly elastic deformation behaviour.

(11) If a larger force is applied to the material, the stress in the material increases. When the stress passes the so-called elastic limit, the atom planes in the material begin to slide over one another. This effect is not reversed if the force is removed and a permanent deformation of the material is therefore achieved. Thus, in this force region the material exhibits a plastic deformation behaviour.

(12) If the force is increased further, the stress will exceed the rupture limit of the material and the material will eventually break.

#### (13) Viscoplastic Materials

(14) Viscoplasticity is a theory that describes materials which behave as solids below a critical values of stress but flows like a viscous liquid at greater values of stress. For metals and alloys, viscoplasticity is the macroscopic behavior caused by a mechanism linked to the movement of dislocations in grains, with superposed effects of inter-crystalline gliding. The mechanism usually becomes dominant at temperatures greater than approximately one third of the absolute melting temperature. Thus, these materials become viscoplastic materials above this critical temperature.

(15) A main difference of viscoplastic materials in comparison to non-viscoplastic material is that the viscoplastic material exhibit a rate-dependent deformation behavior in force region of plastic deformation. That is, the viscoplastic-material does not only deform permanently after the application of the force but continue to undergo a creep flow as a function of time under the influence of the applied force. This creep flow deforms the material further.

#### (16) Viscoelastic Materials

(17) Viscous materials, like glycerine, oil or water, resist shear flow and strain linearly with time when a stress is applied. In pure viscous fluids, deformation is non-recoverable due to rearrangement of molecules. Elastic materials strain when stretched and immediately return to their original state once the stress is removed.

(18) A material is said to be viscoelastic if the material has an elastic (recoverable) part as well as a viscous (nonrecoverable) part. When a load is applied onto a viscoelastic material, the elastic deformation is instantaneous while deformation of the viscous part occurs over time.

(19) Since viscoelastic materials have elements of both elastic (recoverable) character viscous (nonrecoverable) character it is said to exhibit time-dependent strain, i.e. it deforms over time by so called creep. Viscoelastic materials may also be strain rate-dependent, i.e. they deform differently depending on how fast the load is applied.

(20) Examples of viscoelastic materials are polymers, their viscoelastic behaviour may be explained by entanglement and disentanglement processes on a molecular level of polymer chains.

(21) With reference to the above definitions, the present invention relates to a device that extrudes and/or pultrudes a material that do not undergo any recoverable, or just a minimal portion of, i.e. elastic, transitions springing back from a deformed state to a less deformed state, but only to plastically deformable materials. A non-exhaustive list of examples of such materials are; aluminium, ceramics, alloys, etc.

(22) The invention relates to an extrusion and/or pultrusion device for forming a profile product

made from a plastically deformable material and/or viscoplastic material in a production direction, said device comprising: a rotating die, extending in a radial direction and a width direction, having two opposite first and second side walls and an outer circumferential surface extending in the width direction there between, wherein the rotating die comprises a first side portion in connection to the first side wall and a second side portion in connection to the second side wall and a mid-portion extending between the first and second side portions, and a profile definition zone having a longitudinal direction coinciding with the production direction, a height direction and a width direction being perpendicular to the height direction, comprising a through channel comprising a first channel section followed by a second channel section downstream the first channel section with reference to the production direction,

wherein the rotating die is rotatable about an axis extending across the production direction and arranged to allow the outer circumferential surface to, while the rotating die rotates, exert a pressure onto a surface of the material when fed through the profile definition zone, wherein; the first channel section is circumferentially delimited by one or more walls

(23) and wherein the second channel section is circumferentially delimited by the circumferential surface of the rotating die and a channel portion comprising a counter-bearing opposite the rotating die and opposing first and second channel portion side walls between the rotating die and the counter-bearing

(24) The first channel section is configured to plastically deform the material into a master profile having a maximum height at a predetermined feeding rate dependent on material and minimum cross-sectional area with a first maximum height in the first channel section,

(25) and wherein the second channel section is configured to further deform the material into a final profile having a minimum height by the rotating die being configured to apply increasing pressure on the master profile against the counter-bearing when the master profile exits the first channel section,

wherein the rotating die is configured at a minimum distance between the rotating die and the counter-bearing dependent on a maximum allowable pressure applied by the rotating die at the position of that minimum distance, wherein the maximum allowable pressure corresponds to the difference in height of the master profile and the final profile and dependent on pattern in the circumferential surface of the rotating die.

(26) One advantage here is that maximum allowable pressure is controlled in both the first- and second channel which gives the possibility to design the extrusion and/or pultrusion device dependent on material to be processed and process speed. Controlling the maximum allowable pressure dependent on material to be processed allows for a production rate with high quality output and reduces risk for e.g. rupture due to a too high stress on the material and further reduces risk for damaging the device.

(27) According to one example, the rotating die comprises a pattern comprising at least one indentation, wherein the rotating die is configured at a maximum distance between a bottom of the indentation and the counter-bearing dependent on a minimum allowable pressure applied by the rotating die at the position of that maximum distance for achieving plastic deformation of the material in the indentation.

(28) The pattern in the rotating die causes a corresponding opposite pattern in the final profile, i.e. when the pattern in the rotating die comprises an indentation, it causes a corresponding opposite pattern comprising an elevation in the in the final profile.

(29) The rotating die is thus configured at a maximum distance between the rotating die and the counter-bearing dependent on a minimum allowable pressure applied by the rotating die at the position of that minimum distance, wherein the minimum allowable pressure corresponds to the difference in height of the master profile and the final profile and dependent on pattern in the circumferential surface of the rotating die with regard to achieving the plastic deformation desired in the indentation, which minimum pressure depends on the material to be extruded and/or

pultruded.

(30) It should be noted that the pattern in the rotating die could be arranged such that the device simultaneously exhibits the minimum distance and the maximum distance if the pattern is arranged such that the indentation is positioned in the circumferential surface with surrounding portion(s) of the circumferential surface, at least in the width direction, comprising no indentations when facing the counter-bearing. As an alternative the pattern comprises a number of indentations spread in the width direction with such non-indentation portions between them facing the counter-bearing at the same time. Here, the rotating die exerts both maximum pressure in the non-indentation portion due to the minimum distance and minimum pressure in the indentation due to the minimum distance, at least during a short time interval during rotation of the rotating die. The design choice of maximum pressure and minimum pressure further allows for an optimized form change of the master profile to the final profile dependent on material such that the material in the indentation fills the indentation and becomes plastically deformed at the same time as the material between the outermost parts, in the radial direction, of the rotating die does not exceed the maximum pressure allowed for the material and/or the design of the extrusion and/or pultrusion device. It should further be noted that some materials exhibit properties that allows for easy filling of the indentations due to the initial pressure difference between the inside and the outside of the indentation. When the indentation is filled a steady state condition, with regard to pressure difference, is achieved during a short time period. In such steady state condition the pressure in the material is balanced and the pressure difference is minimized. For some materials the pressure could be equal or essentially equal in both the indentation and the surrounding non-indentation part during that part of the operation when the indentation faces the counter bearing and the indentation is filled.

(31) With reference to, but not being bound to, the above definitions, it is believed that a minimum pressure is required to achieve plastic deformation. For example, it is believed that aluminium has a minimum pressure level that depends on inter alia temperature and the higher the temperature the less pressure is required. However, a too high pressure and thus a too high temperature may result in the aluminium liquifying which removes the advantage of plastic deformation. The temperature increase in the material is at least in part achieved in the first channel section where the minimum cross-sectional area forces a form change of the material with pressure on the material in all directions in the first channel section and the form change increase the temperature. In the second channel section, the rotating die continues to change the shape of the master profile into the final profile by exerting a balanced pressure within the minimum and maximum limits. Should the rotating die comprise a pattern with one or more indentations, then the minimum pressure is also vital for filling the indentations in a manner with sufficient pressure to achieve plastic deformation within the indentation. At the same time, that part of the rotating die lacking indentations exerts a higher pressure on the master profile due to a lesser distance between the rotating die and the counter-bearing than between the bottom of the indentation and the counter-bearing. Hence, when the rotating die comprises indentations, i.e. a pattern, then both the minimum pressure and the maximum pressure becomes especially important and thus the respective maximum and minimum distance between the rotating die and the counter-bearing.

(32) In order to more easily explain the device, a cylindrical coordinate system has been used for the rotating die and an orthogonal Cartesian coordinate system for a three-dimensional space for the device in general. The rotating die therefore is described as having a width direction from end to end coinciding with a centre line, i.e. rotation axis, about which the rotating die rotates, and a thickness in a radial direction being orthogonal to the width direction. The outer circumferential surface further extends about the axis in a rotation direction being perpendicular to the width direction. Here, rotation symmetric refers to a symmetrical disposition about the rotating axis or a rotational balanced disposition of the matter in the rotating die. The device in general, i.e. e.g. the profile definition zone, the first and second channel sections, is described as having a width

direction, a height direction and a longitudinal direction, where the longitudinal direction coincides with the general production direction.

(33) The rotating die is arranged to be rotatable about the axis and the axis can be directly or indirectly stored in and rotatably coupled to the first and second channel portion side walls.

(34) With reference to the above-described coordinate systems it should be noted that the axis of the rotating die can be arranged perpendicular to the longitudinal direction, i.e. to the production direction of the device in general, or can be arranged at an angle.

(35) According to one example, the axis of the rotating die is directed substantially perpendicular to the production direction with the outer circumferential surface extending across the production direction in a width direction thereof.

(36) According to one example, the axis of the rotating die coincides with the width direction of the device in general and the width direction of the rotating die coincide with the width direction of the device in general. The longitudinal direction coincides with the production direction, i.e. the main direction along which the material travels during production.

(37) According to one example, the axis of the rotating die does not coincide with the width direction of the device in general, but the axis of the rotating die and the width direction of the rotating die is arranged at an angle being less or more than  $90^\circ$  to the longitudinal direction.

However, the axis of the rotating die is arranged such that the outer circumferential surface extends across the production direction in a width direction thereof.

(38) With reference to either one of the two examples above, the normal to the axis of the rotating die coincides with the height direction of the device in general. Here, the normal coincides with the radial direction of the rotating die. Here, the axis of the rotating die is directed perpendicular to the normal of the production direction regardless of whether the axis of the rotating die coincide or not with the width direction of the device in general. However, according to one example the normal to the axis of the rotating die can be arranged at an angle to the height direction of the device in general. However, the axis of the rotating die is arranged such that the outer circumferential surface extends across the production direction in a width direction thereof, but at an angle to the production direction.

(39) According to one example, the one or more walls define a first cross-section at the end of the first channel section and wherein the second channel section defines a second cross-section at a position where the distance between the circumferential surface and the counter-bearing is at a minimum. As stated above, the geometry of the first channel section is different from the second channel section such that the material passing through the first channel section changes form when entering the second channel section. The changing of form is essential for increasing or maintaining the pressure level to such an extent that it will overcome the internal resistance (shear stresses) of the material, fast enough, for the material to saturate the second cross section, including an imprint of the rotating die.

(40) According to one example, the minimum distance in the height direction between the circumferential surface and the counter-bearing in the second cross-section is less than a maximum distance in the height direction in the first cross-section. This has the advantage that the material entering the second channel section will be compressed in the second channel section such that the pressure is increased or maintained to such level that the material will transform fast enough to saturate the second channel section, including the imprint of the rotating die.

(41) Hence, the pressure is increased or maintained to such level that the material will transform fast enough to saturate the second channel section, including an imprint of the rotating die. The pressure is achieved by a combination of an imprint depth of a pattern in the circumferential surface and a Poisson effect and/or a combination of the shape transition due to the geometrical shape difference between the first and second cross-sections and the Poisson effect.

(42) According to one example, the geometry of the first channel section is different from the second channel section such that the material passing through the first channel section changes



form when entering the second channel section,

(43) wherein the master profile has a first cross-section area geometry corresponding to the first cross-section and wherein the final profile has a second cross-section area geometry defined by the second cross-section, wherein the first cross-section area geometry is different from the second cross-section area geometry in any given comparable position, wherein the maximum pressure and thus the minimum distance in the second channel section is dependent on a difference of cross-section area geometry of the master profile and the cross-section area geometry of the final profile.

(44) This has the advantage that the second channel section can be optimized dependent on the level of material transformation from the master profile to the final profile.

(45) According to one example, the rotating die is configured, before forming the profile product, to alter form during forming of the final profile dependent on the maximum allowable pressure and/or, wherein the counter-bearing is configured, before forming the profile product, to alter form during forming of the profile product dependent on the maximum allowable pressure.

(46) This has the further advantage that the rotating die can be configured to change shape from a start-up procedure to steady state operation conditions due to heat and pressure during the production, giving a predicted shape of the final profile. This has the further advantage that the counter-bearing can be configured to change shape due to heat and pressure during the production, giving a predicted shape of the final profile in a similar manner.

(47) According to one example, the first channel section comprises side walls in the form of a top pre-bearing and an opposing bottom pre-bearing. The top pre-bearing is arranged over the opposing bottom pre-bearing in the height direction and are advantageously positioned in or at least in the vicinity of where the first channel section transitions into the second channel section. One advantage here is that the top pre-bearing and/or the bottom pre-bearing can be optimised to release a certain master profile cross-section geometry into the second channel section.

(48) According to one example, also the top pre-bearing and/or the bottom pre-bearing can be configured in a similar manner as the rotating die and/or the counter-bearing to change form from a start-up procedure to a steady state operation. According to one example, the maximum pressure and thus the minimum distance in the second channel section is dependent on the total feeding rate of material in the first channel section, type of material, and temperature of the material when entering the second channel section.

(49) This has the advantage that the second channel section can be optimized dependent on manufacturing speed and material.

(50) According to one example, the maximum allowable pressure applied by the rotating die at the position of the minimum distance is dependent on friction between the material and the counter bearing in the second channel section.

(51) This has the advantage that the pattern in the final profile can be optimized dependent on shear stress applied from the counter bearing in the second channel section due to friction.

(52) According to one example, the cross-section area of the second channel section is configured to be sized with regard to a shrinking effect of the final profile cooling down to the profile product having a final height.

(53) According to one example, the pattern in the rotating die is configured to be sized with regard to a shrinking effect of the final profile cooling down to the profile product.

(54) According to one example, the rotating die is configured with a pattern with at least one indentation, wherein each indentation comprises a release angle dependent on the radius of the rotating die, the intended pattern in the final profile, the configuration of the counter bearing and travelling speed of the final profile in order to achieve the intended profile product. The release angle in the indentation is arranged with relation to a release angle in a corresponding elevation created in the final profile due to the material being pressed into the cavity. Since the rotating die rotates, the indentation can be arranged with a shape that is different from the shape in the profile product taking into account the rotation and release angles affect the shape of the elevation in the

final profile when the elevation is released from the indentation in the production direction.

(55) This has the advantage that the pattern in the final profile can be optimized dependent on design of the pattern in the rotation die.

(56) According to one example, the device is configured to feed a friction material between the counter-bearing and the final profile and/or configured to feed a friction material between the rotating die and the final profile. One advantage here is that, the friction sheet can be used to define and therefore predict friction between the material in any of the first and/or second channels sections. The friction material can be a solid, liquid or gas and can be introduced into the device in any suitable way. For example, the friction material can be introduced by feeding the material to the rotating die and/or the counter-bearing via a separate channel arranged in connection to the first channel section and/or the second channel section. The friction material can alternatively be introduced to the rotating die before the rotating die rotates into the second channel section, i.e. the friction material travels along with the rotating die to the material in the second channels section. The friction material can alternatively be introduced to the material before entering the first channel section, i.e. the friction material travels with the material to be extruded and/or pultruded. Should the friction material be a solid, it could be introduced as a sheet material or any suitable form that allows for the friction sheet to define friction between the material in any of the first and/or second channels sections. Should the friction material be a liquid, it can be introduced by dripping or injecting the liquid according to any one or a combination of the examples above, but is not limited to the examples. Should the friction material be a gas, it can be introduced by injecting the gas according to any one or a combination of the examples above, but is not limited to the examples.

(57) The material that is fed into the device to form the profile product can be in the form of one homogenous material or a mixture of two or more materials that are blended and or layered. The materials can be blended in different ratios and may be blended into a homogeneous mix or a mix with gradients within the material. One material can be a solid and another material can be mouldable, e.g. stone bits and rubber. The material can also be a layered material comprising two or more layers of same or different materials. The material may comprise one or more strings of solid material that follow through the entire extrusion or pultrusion process, e.g. a wire or another reinforcement material.

(58) According to one example, the width of the first channel section is, at least along a portion of its length and at least along a portion of its height, less than a distance between the two opposite side walls of the rotating die. Hence, the first channel section should be at least smaller in width than a distance between the opposing first and second channel portion side walls in the second channel section. The difference in width between the first channel section and the second channel section depends on features of the first and second side portions and tolerances between the rotating die and the respective opposing first and second channel portion side walls. The width of the first channel section should be less than a distance being the distance between the opposing first and second channel portion side walls minus the sum of tolerances, i.e. the sum of the gap between the rotating die side walls and the respective opposing first and second channel portion side walls in the second channel section. If the first and second side portion comprises flange portions, see below for further explanation, then the width of the first channel section is, at least along a portion of its length and at least along a portion of its height, less than a distance between the two flange portions.

(59) One advantage here is that local pressure reduction is achieved in connection to the first and second outer edge portions due to the geometrical difference in the first and second channel sections.

(60) According to one example, local pressure reduction is achieved in connection to the first and/or second side portions due to the geometrical difference in the first and second channel sections and a wake effect downstream the first channel section in connection to the first and/or second side portions.

- (61) According to one example, the first channel section comprises a third side portion extending in the width direction, wherein the third side portion is arranged in relation to the first side portion such that a pressure in the material to be extruded is less in connection to the first side portion than in connection to the third side portion, and/or
- (62) wherein the first channel section comprises a fourth side portion extending in the width direction, wherein the fourth side portion is arranged in relation to the second side portion such that a pressure in the material to be extruded is less in connection to the second side portion than in connection to the fourth side portion. One advantage is that third and fourth side portions creates a wake effect and thus a local pressure decrease downstream the third and fourth side portions that further decreases the local pressure in the first and second side portions of the rotating die.
- (63) According one example, the first channel section comprises leeward means in connection to the third and/or fourth side portions arranged to decrease the space of the first channel section in the height direction being perpendicular to the width direction.
- (64) According one example the first channel section comprises leeward means in connection to the third and/or fourth side portions arranged to decrease the space of the first channel section in the width direction. A combination of leeward means is also possible.
- (65) According to one example, the leeward means is an elevation facing into the through channel. The elevation can be arranged from top to bottom in the first channel section, or can be arranged as a part or several parts along the distance between the top to bottom of the first channel section. The leeward means are advantageously positioned in connection to the first and second side portions of the rotating die.
- (66) One advantage with the leeward means is that the third and fourth side portions further decreases the local pressure in connection to recesses and/or flange portions in the first and second side portions of the rotating die, see below.
- (67) According to one example, the second channel section is arranged in relation to the first channel section with a predetermined second distance between the radially outermost portion of the circumferential surface of the rotating die and the counter-bearing in the channel portion being less than a predetermined first distance between the most far apart portions of the first channel section taken in a height direction coinciding with the radial direction,
- (68) and/or wherein:
- (69) the second channel section is arranged in relation to the first channel section with a predetermined fourth distance between the innermost narrowest portions of the channel portion in the width direction being greater than a predetermined third distance between side walls in the first channel section taken in the width direction at the exit area from the first channel section.
- (70) One advantage is that the narrower first channel section creates a wake effect with decreased pressure downstream in the first channel section and in connection to the first and second side portions of the rotating die due to that the second channels section is broader.
- (71) According to one example, the circumferential surface comprises a textured portion. The entire circumferential surface can be textured, but as an alternative only a portion can be textured.
- (72) According to one example, the circumferential surface is non-textured or has a micro pattern that leaves only an infinitesimal imprint on the profile product that can be visible or non-visible for the human eye.
- (73) According to one example, the channel portion comprises a second rotating die arranged opposite the first rotating die described above. The second rotating die can either replace the counter-bearing in its entirety or can be a part of a static counter-bearing. The second rotating die can be arranged in a similar way as the above described first rotating die to create same or different patterns on two sides of the profile product. The second rotating die can comprise recesses and/or flange portions that can be arranged to cooperate with recesses and/or flange portions of the first rotating die.
- (74) According to one example, the channel portion comprises a third rotating die arranged at an

angle to the first rotating die. This rotating die replaces the opposing first or second channel portion side wall entirely or partly. The third rotating die can be arranged together with only the first rotating die or together with both the first and second rotating die.

(75) According to one example, the channel portion comprises a fourth rotating die arranged opposite the third rotating die. The third rotating die can be arranged together with only the first rotating die or together with both the first and second rotating die

(76) The third and/or the fourth rotating die(s) can be arranged in a similar way as the above described first rotating die to create same or different patterns on two sides of the profile product. The second rotating die can comprise recesses and/or flange portions that can be arranged to cooperate with recesses and/or flange portions of the first rotating die.

(77) According to one example, two or more rotating dies are synchronised. This has the advantage of feeding the material at the same speed. However, it could be possible to also use non-synchronous rotating dies in order to create friction and/or a special pattern and/or to compensate for material differences, or to achieve curved profiles, that follows a radius instead of a straight line at exit of the rotating die.

(78) In all the above examples it is possible to use a combination of textured and non-textured rotating dies.

(79) The invention also refers to a method for producing a profile product by use of a device according to any one of the preceding claims, wherein the method comprises feeding a material to the first channel section and forming the same into the master profile in the first channel section, feeding the material further to the second channel section and forming the same into the final profile in the second channel section. transforming the final profile into the profile product.

(80) According to one example, the final profile and/or the profile product is stretched for achieving the same distance in the pattern along the production direction, i.e. to achieve equal distance between elevations and/or indentations in the pattern along the production direction.

(81) According to one example, the distance between indentations in the pattern on the rotating die is less than a distance between indentations in the corresponding pattern in the production direction on the profile product, wherein a pulling and stretching device is configured to stretch the final profile and/or the profile product so that high precision in distance between features on profile can be achieved by adjustment stretching

(82) Furthermore, extrusion relates to a process where a material is fed by pressure into the first channel section to be formed in the first and second channel sections. Pultrusion relates to where the material to be formed is fed to the device and drawn through the first and second channel sections. It should be noted that the device can be arranged purely for extrusion or purely for pultrusion or a combination of the two.

(83) Furthermore, profile product refers to a product having a three dimensional form, i.e. length, width and height. The profile product may have a cross-section taken in the width and height plane being similar all along the length or may be different dependent on position in length. The cross-section can have any suitable two dimensional shape, for example, round, oval, elliptical, i.e. two sides, undulating, three or more sides or a combination of the same. One or more sides may be patterned, i.e. textured with one or more patterns. The pattern/texture is created by the rotating die.

(84) It should be noted that the invention can be varied within the scope of the claims and that the examples described above and below should not be seen as limiting for the invention.

(85) For example, the first channel section could be circumferentially delimited by static walls or could be arranged with one or more dynamic walls as long as the material can be extruded or pultruded with the device according to the invention. Static walls has the advantage of being cheap and robust.

(86) According to one example, the first channel portions can be arranged centred with relation to the second channels. This has the advantage that the flow of material entering the second channel is evenly distributed. The first and second side portions can be arranged centred with respect to the

first channel section with the advantage of having a evenly distributed decrease in pressure over the rotating die.

(87) For example, the device could comprise several rotating devices arranged side by side, i.e. the rotating device could comprise two or more rotating devices having a common rotating axel. The different rotating devices could be arranged in separate second channels or could be arranged in a common separate channel. The different rotating devices could have the same or different texture to create same or different patterns on the profile product. The profile product could thus comprise one or more strands of internal profiles running along the production direction and being generated by the different rotating devices. The different strands could be separable into separate products at predetermined separation lines that could coincide with the separation of the different rotating devices. However, one separate rotating die could comprise a pattern/texture that separates similar or different patterns such that the profiled product comprises one or more strands of internal profiles running along the production direction. Also here, the strands could be separable in the profiled product.

(88) According to one example, the rotating die and/or the counter-bearing comprises a cooling device that cools down the material when forming the final profile. This has the advantage that a predetermined temperature of the material is achieved for optimum material properties of the profile product. The material temperature when extruding and/or pultruding can for certain materials be crucial for the quality of the profile product. The temperature is also important due to frictional properties between the material and the rotating die and/or the counter-bearing. The cooling device can, for example, be arranged in the form of cooling circuits with gas or liquid fluid conductors arranged within the rotating die and/or the counter-bearing; and/or external devices cooling down the rotating die and/or the counter-bearing; and/or liquids or gaseous fluids added to the rotating die and/or the counter-bearing; or a combination of such devices or any other suitable cooling devices.

(89) According to one example, the rotating die is configured to be cooled on the surface so that the temperature of the rotating die surface is below a predetermined allowed temperature of the extruded material.

(90) According to one example, the rotating die is cooled on the surface so that the temperature of the rotating die surface is at least 10 degrees Celsius below a glass transition temperature or melting temperature of the material.

(91) According to one example, the rotating die is cooled on the surface so that the temperature of the rotating die surface is at least 50 degrees Celsius below a glass transition temperature or melting temperature of the material, enabling higher speed of extruding.

(92) The device can further be arranged for co-extrusion and/or on-extrusion with one or more inlet channels that connects to the first channel section. Hence, one or more materials could be fed to the first channel section via one channel, but two or more materials can be fed to the first channel section via one inlet channel or a multiple channel inlets. The multiple inlet channels can be the same in number as the number of materials or the multiple inlet channels can be less than the number of materials if two or more materials are fed via one inlet channel.

(93) Two or more materials can be fed to the first channel section and/or the second channel section via one inlet channel or multiple channel inlets. The multiple inlet channels can be the same or more in number as the number of materials or the multiple inlet channels can be less than the number of materials if two or more materials are fed via one inlet channel.

(94) The invention further relates to a co-extrusion device and/or an on-extrusion device comprising an extrusion and/or pultrusion device according to what has been described above. The co-extrusion device and/or an on-extrusion device comprises at least two inlet channels that connects directly or indirectly to the second channel section, wherein each of the at least two inlet channels is configured to feed one or more materials at a predetermined distance upstream from the second channel section or to a marriage point for the at least two inlet channels in connection to

where the first channel section transitions into the second channel section.

(95) One advantage is that co-extrusion and/or on-extrusion allows for manufacturing a layered profile product and/or a profile product with different materials and/or a profile product with a core embedded in a surrounding material, e.g. electrical wire, toothed belt etc.

(96) According to one example, the device according to any of the examples above comprises a pulling and stretching device arranged downstream the second channel section and configured to pull the material in the production direction when exiting the second channel section for transforming the final profile to the profile product.

(97) According to one example, the distance between indentations in the pattern on the rotating die is less than a distance between elevations in the corresponding pattern in the production direction on the profile product, wherein the pulling and stretching device is configured to stretch the final profile and/or the profile product so that high precision in distance between features on profile can be achieved by adjustment stretching. The distance between the indentations in the rotating die is taken in the rotation direction, i.e. along the circumferential surface in the rotation direction.

(98) One advantage is that the pulling and stretching device dynamically can stretch the material in the final profile during its transition to profile product, e.g. in order to obtain an equidistant pattern in the production direction of the profile product. The pulling and stretching device can further be used to guide the final product in the width and/or height direction during its transition from final product to profile product in order to control bending.

(99) The pulling and stretching device can be any type of device that comprises means for gripping the material and means for pulling. According to one example, the pulling and stretching device comprises controlling means for controlling the pulling force applied to the material. The controlling means may comprise sensor(s) and/or may be connected to sensor(s) that supervises the state of the final profile and/or the material during its transition from the final profile to the profile product. The sensor(s) comprises means for sending analog and/or digital information to the controlling means. The information relates to the state of the material and the controlling means is configured to process the information for controlling the pulling and stretching device.

(100) This enables good precision in distance between three dimensional-longitudinal features of profile, e.g. teeth in a pinion rack, making it possible to stretch profile so that high equidistant precision between teethes in a pinion rack or toothed belt becomes possible.

(101) This also enables a possibility to design the rotating die so it results in profile product that deliberately has a slightly shorter distance between three dimensional features, before stretching and gives room for stretching/pulling calibration.

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## Description

### BRIEF DESCRIPTION OF THE DRAWINGS

(1) The invention will below be described in connection to a number of drawings, wherein;

(2) FIG. 1 schematically shows a cross-sectional view from below along section A-A in FIG. 2 of a device according to one example of the invention;

(3) FIG. 2 schematically shows a cross-sectional perspective side view of a device according to the invention;

(4) FIG. 2A schematically shows a similar view as FIG. 2, but including a material processed;

(5) FIG. 2B schematically shows a side view along cross-section B-B in FIG. 1 according to one example;

(6) FIG. 2C schematically shows a side view along cross-section B-B in FIG. 1 according to one example;

(7) FIG. 2D schematically shows a side view along cross-section B-B in FIG. 1 according to one example

- (8) FIG. 3A schematically shows a front view of a rotating die according to one example;
- (9) FIG. 3B schematically shows a perspective view of a rotating die according to one example;
- (10) FIG. 4 schematically shows an enhanced portion of the rotating die and the final profile in any of FIGS. 2B-2D;
- (11) FIG. 4A schematically shows an enhanced portion of a final profile in FIG. 4;
- (12) FIG. 4B schematically shows an enhanced portion of a rotating die in FIG. 4;
- (13) FIG. 5 schematically shows a cross-sectional side view of a device according to one example of the invention
- (14) FIG. 6 schematically shows a perspective back view and outlet of a device according to one example of the invention;
- (15) FIG. 7 schematically shows a cross-section of the master profile taken in a first cross-section of FIG. 5 and a cross-section of the final profile taken in the second cross-section of FIG. 5 according to one example;
- (16) FIG. 8 schematically shows a cross-section of the master profile taken in a first cross-section of FIG. 5 and a first cross-section of the final profile taken in the second cross-section of FIG. 5 when the rotating die has imprinted a pattern in the final profile and a second cross-section of the final profile taken in the second cross-section of FIG. 5 when the rotating die has not imprinted a pattern in the final profile;
- (17) FIG. 9 schematically shows a back view and outlet of an assembly of rotary dies including three rotary dies;
- (18) FIG. 10 schematically shows a perspective view of an assembly according to FIG. 9;
- (19) FIG. 11 schematically shows a back view and outlet of an assembly of rotary dies including four rotary dies;
- (20) FIG. 12 schematically shows a perspective view of an assembly according to FIG. 11, and wherein;
- (21) FIGS. 13-18 schematically show co-extrusion device and/or an on-extrusion device comprising a device according to any one of FIGS. 1-12;
- (22) FIG. 13 schematically shows a cross-section side view of a co-extrusion device and/or an on-extrusion device according one example;
- (23) FIG. 14 schematically shows a perspective view of a co-extrusion device and/or an on-extrusion device according one example;
- (24) FIG. 15 schematically shows a cross-section side view of a co-extrusion device and/or an on-extrusion device according one example;
- (25) FIG. 16 schematically shows a cross-section side view of a co-extrusion device and/or an on-extrusion device according one example where the device comprises two opposing rotating dies;
- (26) FIG. 17 schematically shows a cross-section side view of a co-extrusion device and/or an on-extrusion device according one example comprising one rotating die;
- (27) FIG. 18 schematically shows an example where the first inlet channel conveys a continuous solid material in the form of a wire or the like, and a material to be extruded or pultruded in the first and second channel sections, and where;
- (28) FIG. 19 schematically shows a flow chart of a method for producing a profile product by use of a device according to what has been described in connection to FIGS. 1-18.

#### DETAILED DESCRIPTION

- (29) The invention will below be described in connection to a number of drawings. Same features will be denoted with like numbers in all the drawings.
- (30) Here, front view with inlet and back view with outlet are used as an orientation for the reader with regard to production direction where material to be worked is inserted into the inlet and a profile product is shaped in the device and then exits the device via the outlet.
- (31) In some drawings, the production direction is denoted PD with an arrow pointing in the production direction.

(32) FIG. 1 schematically shows a view from below along section A-A in FIG. 2, i.e. in a height direction Z, of a device according to one example of the invention and FIG. 2 schematically shows a cross-section perspective view of the device in FIG. 1. FIGS. 1 and 2 show an extrusion- or pultrusion device **1** for extrusion- or pultrusion in a production direction Y of a profile product **2**, see FIGS. 2A-2D and FIGS. 4, 4A, 4B, 7 and 8, made from a plastically deformable material and/or viscoplastic material. The device comprises: a rotating die **3**, extending in a radial R direction and a width direction X, having two opposite first and second side walls **5**, **6** and an outer circumferential surface **4** extending in the width direction X there between, wherein the rotating die **3** comprises a first side portion **23** in connection to the first side wall **5** and a second side portion **25** in connection to the second side wall **6** and a mid-portion **22** extending between the first and second side portions **23**, **25**, and a profile definition zone **7** having a longitudinal direction Y coinciding with the production direction Y, a height direction Z and a width direction X being perpendicular to the height direction Z, comprising a through channel **8** comprising a first channel section **9** followed by a second channel section **10** downstream the first channel section **9** with reference to the production direction, wherein the rotating die **3** is rotatable about an axis extending across the production direction Y and arranged to allow the outer circumferential surface **4** to, while the rotating die **3** rotates, exert a pressure onto a surface of the material when fed through the profile definition zone **7**, and wherein; the first channel section **9** is circumferentially delimited by one or more walls **11** and wherein the second channel section **10** is circumferentially delimited by the circumferential surface **4** of the rotating die **3** and a channel portion **13** comprising a counter-bearing **14**, shown in FIG. 2, opposite the rotating die **3** and opposing first and second channel portion side walls **15**, **16** between the rotating die **3** and the counter-bearing **14**.

(33) According to an example embodiment, FIGS. 1, 2, 2A-2D schematically show that the first channel section **9** is configured to deform the material into a master profile **36** having a maximum height H1 at a predetermined feeding rate dependent on material and minimum cross-sectional area with a first maximum height D1 in the first channel section **9**,

(34) and wherein the second channel section **10** is configured to further deform the material into a final profile **37** having a minimum height H2 by the rotating die **3** being configured to apply increasing pressure on the master profile **36** against the counter-bearing **14** when the master profile **36** exits the first channel section **9**, wherein the rotating die **3** is configured at a minimum distance D2 between the rotating die **3** and the counter-bearing **14** dependent on a maximum allowable pressure applied by the rotating die **3** at the position of that minimum distance D2, wherein the maximum allowable pressure corresponds to the maximum difference in height of the master profile **36** and the final profile **37** and dependent on pattern **38** in the circumferential surface **4** of the rotating die **3**.

(35) One advantage here is that maximum load is controlled in both the first- and second channel section which gives the possibility to design the extrusion and/or pultrusion device dependent on material to be processed and process speed. Controlling the maximum load dependent on material to be processed allows for a production rate with high quality output and reduces risk for e.g. rupture due to a too high stress on the material.

(36) FIG. 2A schematically shows a similar view as FIG. 2 and that the material is formed in the first channel section **9** into the master profile **36** and directly thereafter formed in the second channel section **10** into the final profile **37**. FIG. 2A also shows that the final profile **37** has a lesser height H1 than the height H2 of the master profile **36** due to further pressure on the master profile **36** in the second channel section **10**. FIG. 2A also shows that the product profile **2** has a lesser height H3 than the height H1 of the final profile **36** due to shrinking when cooling down from the final profile **37** to the product profile **2**. In FIG. 2A the rotating die lacks indentions **38** as shown in FIGS. 2B-2D. Comparing FIGS. 2B-2D with FIG. 2A, the rotating die **3** in FIG. 2A could be seen, as shown in FIG. 2B, as being rotated such that the part of the rotating die **3** pressing against the material against the counter-bearing **14** lacks indentations **38** and thus exerts maximum pressure



against the material.

(37) FIG. 2B schematically shows a side view along cross-section B-B in FIG. 1 and is similar to FIG. 2A, but the rotating device is rotated such that the part of the rotating die 3 pressing against the material against the counter-bearing 14 does not comprise an indentation 38. FIGS. 2C and 2D show a similar device 1 as FIG. 2B, but where the rotating die 3 is rotated such that the indentation 38 with a bottom 44 faces the counter bearing 14.

(38) FIG. 2B schematically shows an initial zone A where the material is pressed by a device (not shown) into the first channel section 9 either by a device (not shown) exerting an external pressure in the production direction, i.e. extrusion, and/or by the material being dragged through the first channel 9 by a device (not shown) dragging the material in the production direction PD, i.e. pultrusion. Zone A of the device comprises a funnel shaped opening 43 where the material changes form from an initial form having a larger cross-section than the first channel section 9. The shape of the opening can, however, vary depending on material, temperature and device pressing the material.

(39) FIG. 2B schematically shows a zone B arranged directly after zone A, wherein zone B corresponds to the first channel section 9 and where the formation of the master profile 36 takes place due to the material changing form due to the pressure exerted on the material from the side walls 11 in the first channel section 9 when the material moves through the first channel section 9.

(40) FIG. 2B schematically shows a zone C arranged directly after zone B, wherein zone C corresponds to the second channel section 10 and where the formation of the final profile 36 takes place due to the material changing form due to the pressure exerted on the material from at least the rotating die 3 and the opposing counter bearing 14 in the second channel section 10 when the material moves through the second channel section 9.

(41) FIG. 2B schematically shows a zone D arranged directly after zone C, wherein zone D corresponds to a section of the production line after the second channel section 10 and where the material starts to cool down and where the final profile 36 starts to change form due to shrinking as a consequence of the temperature drop. In zone D, the final profile 37 can be subject to various production measures for achieving desired material properties. For example, cooling, heating, stretching, compressing, etc. in order to change the final profile 37 into a profile product 2 with desired material properties.

(42) The length of Zone D depends on material properties and a working environment surrounding the material in zone D. The material properties are e.g. heat dissipation and the mass of the material to be cooled down. For example, a thinner material cools down faster than a thicker material. The working environment refers e.g. to ambient temperature and humidity. For example, a warmer environment slows down the cooling process compared to a cooler environment.

(43) FIG. 2B schematically shows a zone E arranged after zone D, wherein zone E corresponds to a section of the production line where the material has cooled down to a predetermined temperature representing a temperature establishing the final form of the product profile and where no or an infinitesimal change of form will continue. FIG. 2B, shows that the profile product has a height H3 in Zone E being less than the height H2 of the final profile 37. In the same manner, the pattern 39 of the final profile 37 has shrunk to a pattern 40 in zone E due to the cooling.

(44) FIG. 2B shows an example, where the device 1 according to any of the examples above comprises a pulling and stretching device 54 arranged downstream the second channel section 10 and configured to pull the material in the production direction PD when exiting the second channel section 10 for transforming the final profile 37 to the profile product 2.

(45) One advantage is that the pulling and stretching device dynamically can stretch the material in the final profile during its transition to profile product, e.g. in order to obtain an equidistant pattern in the production direction of the profile product. The pulling and stretching device can further be used to guide the final product in the width and/or height direction during its transition from to final product to profile product in order to control bending.

(46) According to one example, the distance between indentations **38** in the pattern **38** on the rotating die **3** is less than a distance between elevations **40** in the corresponding pattern **38** in the production direction on the profile product **2**, wherein the pulling and stretching device **54** is configured to stretch the final profile **37** and/or the profile product **2** so that high precision in distance between features on profile can be achieved by adjustment stretching.

(47) The pulling and stretching device can be any type of device that comprises means for gripping the material and means for pulling. According to one example, the pulling and stretching device comprises controlling means **55** for controlling the pulling force applied to the material. The controlling means **55** may comprise sensor(s) and/or may be connected to sensor(s) **56** that supervises the state of the final profile and/or the material during its transition from the final profile to the profile product. The sensor(s) comprises means for sending analog and/or digital information to the controlling means. The information relates to the state of the material and the controlling means is configured to process the information for controlling the pulling and stretching device. In FIG. 2B, the rotating die **3** and/or the counter-bearing **14** comprises a cooling device **57** that cools down the material when forming the final profile **37**. This has the advantage that a predetermined temperature of the material is achieved for optimum material properties of the profile product. The material temperature when extruding and/or pultruding can for certain materials be crucial for the quality of the profile product. The temperature is also important due to frictional properties between the material and the rotating die and/or the counter-bearing. The cooling device can, for example, be arranged in the form of cooling circuits with gas or liquid fluid conductors arranged within the rotating die and/or the counter-bearing; and/or external devices cooling down the rotating die and/or the counter-bearing; and/or liquids or gaseous fluids added to the rotating die and/or the counter-bearing; or a combination of such devices or any other suitable cooling devices. It should be noted, that rotating die **3** can be configured to operate without the cooling device **57** in FIG. 2B, as is shown in e.g. FIGS. 2C-2D.

(48) According to one example, the rotating die **3** is configured to be cooled on the surface so that the temperature of the rotating die surface is below a predetermined allowed temperature of the extruded material.

(49) According to one example, the rotating die is cooled on the surface so that the temperature of the rotating die surface is at least 10 degrees Celsius below a glass transition temperature or melting temperature of the material.

(50) According to one example, the rotating die is cooled on the surface so that the temperature of the rotating die surface is at least 50 degrees Celsius below a glass transition temperature or melting temperature of the material, enabling higher speed of extruding.

(51) FIGS. 2B-2D schematically show that the rotating die **3** comprises a pattern **38** comprising at least one indentation **38** in the circumferential surface **4**. In FIGS. 2B-2D, the pattern **38** comprises 4 indentations, but the number of indentations is here only an illustrative example and there may be more or less indentation in a pattern spread over the rotating die in a predetermined design depending on the desired features of the profile product **2**. The indentations can have any shape suitable, for example oval, round, polygon or a mixture of such or other shapes. The indentations **38** have a bottom **44** at a maximum depth of the indentation and the indentations may have different or similar depths. Between the indentations **38** the rotating die comprises portions that have a minimum distance **D2** between the rotating die **3** and the counter bearing **14** when facing the counter-bearing **14**. The indentation **38** with the largest distance between the bottom **44** and the counter-bearing **14** when facing the counter-bearing forms a maximum distance **D22**, see FIGS. 2C and 2D, between the rotating die **3** and the counter-bearing **14**.

(52) According to one example, the minimum distance **D2** in the height direction **Z** between the circumferential surface **4** and the counter-bearing **14** in the second cross-section **17** is less than a maximum distance **D1** in the height direction in the first cross-section **12**.

(53) FIGS. 2C and 2D schematically shows the same side view as FIG. 2B with Zones as described

above, but with the rotating die **3** being rotated such that one indentation **38** faces the counter-bearing **14**.

(54) FIGS. **2C** and **2D** schematically show that the rotating die **3** is configured at a maximum distance **D22** between a bottom **44** of the indentation **38** and the counter-bearing **14** dependent on a minimum allowable pressure applied by the rotating die at the position of that maximum distance **D22** for achieving plastic deformation of the material in the indentation **38**.

(55) FIGS. **2B-2D**, schematically shows that, the pattern **38** in the rotating die **3** is configured to be sized with regard to a shrinking effect of the final profile **37** cooling down to the profile product **2**.

(56) It should be noted that the pattern **38** in the rotating **3** could be arranged such that the device **1** simultaneously exhibits the minimum distance **D2** and the maximum distance **D22** if the pattern **38** is arranged such that the indentation is positioned in the circumferential surface **4** with surrounding portion(s) of the circumferential surface **4**, at least in the width direction **X**, comprising no indentations when facing the counter-bearing **14**. As an alternative the pattern **38** comprises a number of indentations **38** spread in the width direction **X** with such non-indentation portions between them facing the counter-bearing **14** at the same time. Here, the rotating die **3** exerts both maximum pressure in the non-indentation portion due to the minimum distance **D2** and minimum pressure in the indentation due to the minimum distance **D22** during at least a short time interval. The design choice of maximum pressure and minimum pressure further allows for an optimized form change of the master profile to the final profile dependent on material such that the material in the indentation fills the indentation and becomes plastically deformed at the same time as the material between the outermost parts, in the radial direction, of the rotating die does not exceed the maximum pressure allowed for the material and/or the design of the extrusion and/or pultrusion device. It should further be noted that some materials exhibit properties that allows for easy filling of the indentations due to the initial pressure difference between the inside and the outside of the indentation. When the indentation is filled a steady state condition, with regard to pressure difference, is achieved during a short time period. In such steady state condition the pressure in the material is balanced and the pressure difference is minimized. For some materials the pressure could be equal or essentially equal in both the indentation and the surrounding non-indentation part during that part of the operation when the indentation faces the counter bearing and the indentation is filled.

(57) It should also be noted that the rotating die can have a different circumferential distance between the indentations **38** than what is given in the finished profile product should it be rolled up around the rotating die, allowing for compensation and adjusting distance between features to get high precision on finished profile product by e.g. stretching the final profile after extrusion.

(58) FIG. **2C** schematically shows that an upper portion **41**, also called pre-bearing **41** below, of the wall **11** in the first channel section **9** is arranged at a height level in the **Z**-direction above a maximum height level of the bottom **44** of the indentation **38** when the indentation **38** faces the counter-bearing **14**. FIG. **2D** schematically shows that the upper portion **41** of the wall **11** in the first channel section **9** is arranged at a height level in the **Z**-direction below a maximum height level of the bottom **44** of the indentation **38** when the indentation **38** faces the counter-bearing **14**. The height level of the upper portion **41** can vary dependent on material and on the pattern **38** in the rotating die **3** and thus how the material changes form to fill the indentation **38** with the plastic deformation.

(59) FIGS. **2A**, **2B**, **2C** and **2D** schematically show that the first channel section **9** comprises side walls **11** in the form of a top pre-bearing **41** and an opposing bottom pre-bearing **42**. The top pre-bearing **41** is arranged over the opposing bottom pre-bearing **42** in the height direction **Z**.

(60) According to one example, the cross-section area **A1** of the second channel section **10** is configured to be sized with regard to a shrinking effect of the final profile **37** cooling down to the profile product **2** having a final height **H3**.

(61) According to one example embodiment, FIG. **1** schematically shows that the width **D3** of the

first channel section **9** is, at least along a portion of its length and at least along a portion of its height, less than a distance **D4** between the two opposite side walls **5**, **6** of the rotating die **3**. Hence, the first channel section **9** should be at least smaller in width than a distance between the opposing first and second channel portion side walls **15**, **16** in the second channel section **10**. The difference in width between the first channel section **9** and the second channel section **10** depends on features of the first and second side portions **23**, **25** and tolerance between the rotating die **3** and the respective opposing first and second channel portion side walls **15**, **16**. The width **D3** of the first channel section **9** should be less than a distance **D4** being the distance between the opposing first and second channel portion side walls **15**, **16** minus the sum of tolerances, i.e. the sum of the gap between the rotating die side walls **5**, **6** and the respective opposing first and second channel portion side walls **15**, **16** in the second channel section **10**. If the first and second side portion comprises flange portions **18**, **19**, see below for further explanation, then the width **D3** of the first channel section **9** is, at least along a portion of its length and at least along a portion of its height, less than a distance **D4** between the two flange portions **18**, **19**.

(62) One advantage is that a local pressure reduction is achieved in connection to the first and second outer edge portions **5**, **6** due to the geometrical difference in the first and second channel sections **9**, **10**. The local pressure reduction reduces the flow speed of the material and this removes leakage problems between the first side wall **5** and the first and the first channel portion side wall **15**; and between the second side wall **6** and the second channel portion side wall **16**. This will be explained further below and also in combination with additional leakage protection strategies. FIG. **1** shows one example of an additional and FIG. **5** shows another example of leakage protection strategies. The different examples can be combined, which will be explained further below.

(63) It should be noted that the rotating die **3** can be cylindrical or non-cylindrical and textured or not textured dependent on desired profile of the profile product.

(64) According to one example shown in FIGS. **3A** and **3B**, the rotating die **3** may be configured, before forming the profile product **2**, to alter form during forming of the final profile **37** dependent on the maximum allowable pressure. In FIGS. **3A** and **3B**, at least a mid-portion **22** of the circumferential surface **4** of the rotating die **3** is convex in order to allow bending due to, at least, the forces and temperature during steady state production so that a predicted pattern **39** can be achieved in the final profile **37** during the steady state operation. Here, steady state operation refers to stable operating conditions after a start-up procedure.

(65) According to one example (not shown), the counter-bearing **14** is configured, before forming the profile product **2**, to alter form during forming of the profile product **2** dependent on the maximum allowable pressure. According to one example (not shown), also the top pre-bearing **41** and/or the bottom pre-bearing **42** can be configured in a similar manner to change form a start-up procedure to a steady state operation.

(66) According to one example shown in FIGS. **4**, **4A** and **4B**, the rotating die **3** is configured with a pattern **38** with at least one indentation **38** with a bottom **44**, wherein each indentation **38** comprises a release angle **a2** compared to a release angle **a1** of the corresponding elevation in the pattern **39** in the final profile **37**. The release angles **a1** and **a2** are dependent on the radius of the rotating die **3**, the intended pattern **39** in the final profile **37**, the configuration of the counter bearing and travelling speed of the final profile **37**. However, **a1** is greater than **a2** dependent on radius of the rotating die **3** and type of pattern **38** in the rotating die **3**.

(67) FIGS. **4**, **4A** and **4B** show that the indentation **38** in the pattern **38** has a height **D23**, alternatively called depth, in the radial direction **R** from the bottom **44** of the indentation **38** to the indentation delimiting portion in the circumferential surface **4** of the rotating die **3**. The height **D23** plus the distance **D2** in FIG. **2B** is equal to the maximum distance **D22** in FIGS. **2C** and **2D** and is thus relevant for the minimum allowable pressure for achieving plastic deformation of the material in the indentation **38**. FIG. **4A** further shows that the pattern **39** in the final profile has an elevation with a height **H4** that correspond to the depth **D23** of the indentation **38**.

(68) In FIG. 1 the first side portion **23** comprises a first flange portion **18** extending in a radial direction R with an extension exceeding the radial extension of at least a part of the mid-portion **22** and wherein the second side portion **25** comprises a second flange portion **19** extending in the radial direction with an extension exceeding the radial extension of at least a part of the mid-portion **22**.

(69) The first flange portion **18** and the second flange portion **19** are arranged to prevent movement of the material outside the rotating die **3** in a direction towards the opposing first and second channel portion side walls **15**, **16**.

(70) The rotating die **3** can be arranged without flange portions which is shown at least in FIGS. 2B-2D, **4**, **4A** and **4B**.

(71) FIG. 5 schematically shows a cross-sectional side view of a device according to one example embodiment of the invention and FIG. 6 schematically shows a back view and outlet of the device in FIG. 5. In FIG. 5, the one or more walls **11** define a first cross-section **12** at the end of the first channel section **9** and wherein the second channel section **10** defines a second cross-section **17** at a position where the distance between the circumferential surface **4** and the counter-bearing **14** is at a minimum, and wherein the geometry of the first channel section **9** is different from the second channel section such that the material passing through the first channel section **9** changes form when entering the second channel section **10**.

(72) FIG. 7 schematically shows a cross-section of the master profile **36** taken in the first cross-section **12** of FIG. 5 and a cross-section of the final profile **37** taken in the second cross-section **17** of FIG. 5 according to one example; Referring also to FIGS. 1-2D, FIG. 7 schematically shows a cross-section of the master profile **36** and the final profile **37** when using a non-textured rotating die, i.e. no pattern **38** with indentations **38** as described above.

(73) FIG. 8 schematically shows a cross-section of the master profile **36** taken in the first cross-section **12** of FIG. 5 and a first cross-section of the final profile **37** taken in the second cross-section **17** of FIG. 5 when the rotating die **3** has imprinted a pattern **39** in the final profile and a second cross-section of the final profile **37** taken in the second cross-section **17** of FIG. 5 when the rotating die **3** has not imprinted a pattern **39** in the final profile;

(74) Referring also to FIGS. 1-2D, FIG. 8 schematically shows a cross-section of the master profile **36** and the final profile **37** when using a textured rotating die, i.e. with a pattern **38** with indentations **38** as described above. The pattern **38** in the rotating die causes a corresponding opposite pattern **39** in the final profile, i.e. when the pattern **38** in the rotating die **3** comprises an indentation **38**, it causes a corresponding opposite pattern **39** comprising an elevation **39** in the final profile **37**. In FIG. 8 the cross-section of the final profile **37** has been denoted **37a** when the final profile has an elevation **39** caused by the indentation **38**. In FIG. 8 the cross-section of the final profile **37** has been denoted **37b** when the final profile lacks an elevation **39** caused by the circumferential surface **4** of the rotating die **3** in the space between the indentations **38**.

(75) Referring to FIGS. 1-2D and FIGS. 7 and 8, the master profile **36** has a first cross-section area geometry **A1** corresponding to the first cross-section **12** and wherein the final profile **37** has a second cross-section area geometry **A2** defined by the second cross-section **17**, wherein the first cross-section area geometry **A1** is different from the second cross-section area geometry **A2** in any given comparable position, wherein the maximum pressure and thus the minimum distance **D2** in the second channel section **10** is dependent on a difference of cross-section area geometry **A1** of the master profile **36** and the cross-section area geometry **A2** of the final profile **37**.

(76) FIG. 5 shows that the walls **11** are static and define a first cross-section **12** at the end of the first channel section **9** and wherein the second channel section **10** defines a second cross-section **17** at a position where the distance **D2** between the circumferential surface **4** and the counter-bearing **14** is at a minimum, and wherein the geometry of the first channel section **9** is different from the second channel section **10** such that the material passing through the first channel section **9** changes form when entering the second channel section **10**.

(77) The minimum distance **D2** in the height direction **Z** between the circumferential surface **4** and the counter-bearing **14** in the second cross-section **17** is less than a maximum distance **D1** in the height direction in the first cross-section **12**. This has the advantage of forcing the material to change form and start flowing in various directions dependent on the shape and form of the rotating die **3** and shape and form of the counter-bearing **14** opposite the rotating die **3**.

(78) Due to the geometrical difference in the first channel section **9** and the second channel section **10**, the pressure in the second channel section **10** is increased or maintained to such level that the material will transform fast enough to saturate the second channel section, including an imprint of the rotating die.

(79) The change of geometry in the master profile **36** and final profile **37** in FIGS. **7** and **8** is relevant for all examples discussed above and below. It should be noted that the first channel section **9** and the second channel section **10** can be formed with different cross-section geometries such as oval, round, polygon, undulating or a combination of one or many shapes.

(80) With reference to FIG. **1-2D**, the second channel section **10** is advantageously arranged in relation to the first channel section **9** with a predetermined second distance **D2**, shown in FIG. **5**, between the radially outermost portion of the circumferential surface **4** of the rotating die **3** and the counter-bearing **14** in the channel portion **13** being less than a predetermined first distance **D1**, shown in FIG. **5**, between the most far apart portions of the first channel section **9** taken in a height direction **Z** coinciding with the radial direction, and/or wherein:

(81) the second channel section **10** is arranged in relation to the first channel section **9** with a predetermined fourth distance **D4**, shown in FIG. **1**, between the innermost narrowest portions of the channel portion **13** in the width direction **X** being greater than a predetermined third distance **D3**, shown in FIG. **1**, between side walls in the first channel section taken in the width direction **X** at the exit area from the first channel.

(82) This change in both height and width forces the material to reform and the narrower first channel section gives a locally decreased pressure when entering the channel section since the first and second side portions are in the wake, i.e. behind the side walls in the first channel.

(83) Furthermore, with reference to FIG. **1** the first and second side walls **5**, **6** are positioned in relation to the first and second channel portion side walls **15**, **16** such that the first and second side walls **5**, **6** are rotatably connected to the first and second channel portion side walls **15**, **16** with a tolerance arranged dependent on product material and the geometrical relation between the first and second channel sections **9**, **10**

(84) The circumferential surface **4** may comprise a textured portion **30** that can cover all the rotating die but the annular recess portion, or the first side portion **4** comprises a non-textured portion **31** extending between the first flange portion **18** and the textured portion **30** and wherein the second side portion **25** comprises a non-textured portion **32** between the second flange portion **19** and the textured portion **30**.

(85) The non-textured portions **31**, **32** advantageously has a radius less than a radius to an imprint depth of the textured portion **30**, especially in the annular recess **19** portion. However according to one example (not shown), the circumferential surface **4** can be non-textured but with a smooth surface or a micro-patterned surface. The non-textured rotating die can have a shape being cylindrical or undulating.

(86) FIG. **9** schematically shows a back view and outlet of an assembly of rotary dies **3** including three rotary dies, **3**, **33**, **34** and FIG. **10** schematically shows a perspective view of an assembly according to FIG. **9**. With references to FIGS. **1**, **5** and **6**, the channel portion **13** comprises a second rotating die **33** arranged opposite the first rotating die **3** replacing the counter-bearing **14** in FIGS. **1**, **5** and **6**. The second rotating **33** die can either replace the counter-bearing **14** in its entirety or can be a part of a static counter-bearing **14** (not shown). The second rotating **33** die can be arranged in a similar way as the above described first rotating die **3** to create same or different patterns on two sides of the profile product. The second rotating die **33** can comprise annular

recesses and/or flange portions that can be arranged to cooperate with annular recesses **29** and/or flange portions **18**, **19** of the first rotating die **3**.

(87) According to one example shown in FIGS. **9** and **10**, the channel portion **13** (shown in FIGS. **1**, **5** and **6**) comprises a third rotating die **34** arranged at an angle to the first rotating die. This rotating die replaces the opposing first or second channel portion side wall **15**, **16** entirely or partly. The third rotating die **34** can be arranged together with only the first rotating die or together with both the first and second rotating die. Hence, the above described arrangement with a first rotating die **3** and a second opposing rotating die **33** can be assembled without the third rotating die **34**.

(88) FIG. **11** schematically shows a back view and outlet of an assembly of rotary dies including four rotary dies, and wherein FIG. **12** schematically shows a perspective view of an assembly according to FIG. **11**. FIGS. **11** and **12** show that the channel portion **13** (shown in FIGS. **1**, **5** and **6**) comprises a fourth rotating die **35** arranged opposite the third rotating die **34**. The fourth rotating die **34** can as an alternative be arranged together with only the first rotating die **3** or together with both the first and second rotating die **3**, **33**.

(89) The third and/or the fourth rotating die(s) **34**, **35** can be arranged in a similar way as the above described first rotating die **3** to create same or different patterns on two sides of the profile product. The third and/or fourth rotating dies **34**, **35** can comprise annular recesses and/or flange portions that can be arranged to cooperate with annular recesses **29** and/or flange portions **18**, **19** of the first rotating die **3**.

(90) According to one example, two or more rotating dies are synchronised. This has the advantage of feeding the material at the same speed. However, it could be possible to also use non-synchronous rotating dies in order to create friction and/or a special pattern and/or to compensate for material differences.

(91) The device can be arranged with a combination of textured and non-textured rotating dies **3**; **33**; **34**; **35**.

(92) FIGS. **13-19** schematically show a co-extrusion device **1a** and/or an on-extrusion device **1a** comprising an extrusion and/or pultrusion device **1** according to any one of the examples discussed above, wherein the device **1a** comprises at least two inlet channels **45**, **46**, **47** that connects directly or indirectly to the second channel section **10**, wherein each of the at least two inlet channels **45**, **46**, **47** is configured to feed one or more materials at a predetermined distance upstream from the second channel section **10** or to a marriage point for the at least two inlet channels **45**, **46**, **47** in connection to where the first channel section **9** transitions into the second channel section **10**.

(93) Here, co-extrusion refers to where at least two material streams are together processed and formed into the master profile and then into the final profile or where the at least two material streams are together processed and formed into the final profile. Here, on-extrusion refers to where the at least two material streams are positioned in a layered fashion either by being together processed and formed into the master profile and then into the final profile or by bringing together the at least two material streams into the master profile at the marriage point and thereafter together processing and forming the joint at least two material streams into the final profile in the second channel section **10**.

(94) FIG. **13** schematically shows a cross-section side view of a device **1**, **1a** according to the invention. FIG. **13** shows that the profile definition zone **7** comprises a first inlet channel **45** in the form of the first channel section **9** and a second inlet channel **46** in the form of a third channel section **46** connected to the profile definition zone **7** upstream the second channel section **10** for feeding an additional material to the second channel section **10** for forming a layered profile product **2** with material from the first channel section **9**.

(95) According to one example, the third channel section **46** is an extrusion- or pultrusion channel similar to the first channel section **9** arranged to work the material. According to one example, the third channel section **46** is a third channel section that **46** is configured as a conveyer unit for conveying a material to the profile definition zone **7**.

(96) FIG. 14 schematically shows a perspective view of a device according to invention and FIG. 15 schematically shows a cross-section side view of a device according to the invention. FIGS. 13-15 show in different ways that the device 1, la comprises one rotating device 3 as described above and two material streams that are brought together via the first and third channel sections 9, 46.

(97) FIG. 16 schematically shows a cross-section side view of a device according to the invention where the device comprises two opposing rotating dies 3, 33. FIG. 16 further shows that the device 1a comprises a first inlet channel 45 in the form of the first channel section and a second inlet channel 46 in the form of a third channel section 46 connected to the profile definition zone 7 upstream the second channel section 10 for feeding an additional material to the second channel section 10 for forming a layered profile product 2 with material from the first channel section 9. FIG. 16 further shows that the device comprises a third inlet channel 47 in the form of a fourth channel section 47 for feeding a third material to the profile definition zone 7.

(98) According to one example, the fourth channel section 47 is an extrusion- or pultrusion channel similar to the first channel section 9 arranged to work the material. According to one example, the fourth channel section 47 is a fourth channel section that is configured as a conveyer unit for conveying a material to the profile definition zone 7.

(99) FIG. 17 schematically shows a cross-section side view of a device 1 according to the invention comprising one rotating die 3 and a first, third and fourth channel section 9, 44, 45 for feeding three different materials to the profile definition zone 7 according to what is discussed in connection to FIG. 16.

(100) It should be noted that in FIGS. 13-17, the first inlet channel 45 may be the first channel section 9 or an inlet channel 45 that transports material to the first channel section 9.

(101) FIG. 17 shows an example where the first inlet channel 45 conveys a solid material 50, e.g. a wire or a the like, to the first channel section 9 and where the second and third inlet channels 46, 47 introduces one or more materials to be extruded or pultruded in the first channel section 9 and the second channel section 10. The one or more materials may be layered onto the solid material or may surround the solid material.

(102) FIG. 18 schematically shows an example where the first inlet channel 45 conveys a continuous solid material in the form of a wire or the like, and a material to be extruded or pultruded in the first and second channel sections 9, 10. In FIG. 18, the first inlet channel 45 and the second inlet channel 46 are arranged such that the material from the second inlet channel 46 surrounds the solid material 50 and embeds the solid material 50. In FIG. 18, the second inlet channel 46 comprises a pressurized chamber 51 upstream the first channel section 9 for forming the material around the solid material 50. The pressurized chamber 51 comprises a back wall 52 delimiting the pressurized chamber. The second inlet channel 46 comprises a feeding channel 53 to the pressurized chamber 51 for feeding the material to the chamber 51. The back wall 52 comprises the first inlet channel 45 that conveys the solid material 50 and acts as a stop for the material in the chamber to leak through the first inlet channel 45. Here, pressurized means that the material in the second inlet channel 46 is subject to pressure by the material being forced into the second inlet channel 46 and deformed in a similar way as described above with relation to the first channel section 9. In FIG. 18, the first and second channel sections 9, 10 plastically deforms the material in a similar manner as described above. Plastic deformation may take place also in the pressurized chamber, but is not limited to such deformation. Hence, the material in the pressurized chamber may be formed to surround the solid material without being plastically deformed.

(103) With reference to FIGS. 13-19, the different materials are brought together before the second channel section 10 and is then worked in the second channel section as described above. The invention is not limited to three inlet channels 45, 46, 47 or three channel sections 9, 46, 47, but further inlet channels and channel sections are possible in order to manufacture a profile product with same or different materials in different layers.

(104) According to any one of the preceding examples, the material that is fed into the device to



form the profile product is either one homogenous material or a mixture of two or more materials that are blended and or layered. The materials can be blended in different ratios and may be blended into a homogeneous mix or a mix with gradients within the material. One material can be a solid and another material can be mouldable, e.g. stone bits and rubber. The material can also be a layered material comprising two or more layers of same or different materials. The material may comprise one or more strings of solid material that follow through the entire extrusion or pultrusion process, e.g. a wire or another reinforcement material being surrounded by the plastically deformable material.

(105) Here, solid material refers to a material that does not undergo any deformation in the profile definition zone. A non-exhaustive list of examples of solid materials are; bendable wire, stiff rod-like element, mesh of metal and/or fabric and/or composite and/or other suitable materials, a combination of such solid materials, etc.

(106) According to one example, the maximum allowable pressure applied by the rotating die **3** at the position of the minimum distance **D2** is dependent on friction between the material and the counter bearing **14** in the second channel section **10**.

(107) According to one example, the device **1** is configured to feed a friction material **48** between the counter-bearing **14** and the final profile **37** and/or configured to feed a friction material **48** between the rotating die **3** and the final profile **37**.

(108) According to one example, the friction material **48** is conveyed by the first and/or the second and/or the third inlet channels **45**, **46**, **47** at least during start-up of the device in order to control friction in connection to the rotating die **3** and/or the counter bearing **14**. According to one example, the friction material **18** is conveyed by the first and/or the second and/or the third inlet channels **45**, **46**, **47** during a part of or the entire manufacturing process in order to control friction in connection to the rotating die **3** and/or the counter bearing **14**.

(109) According to one example, the friction material **48** is fed directly to the rotating die **3** such that the friction material rotates with the rotating die from a position before the second channel section **10** to the second channels section. FIG. **2A** schematically shows that the device **1** comprises an external friction material **48** feeding device **49** feeding the friction material **48** to the rotating die **3**. As stated above, the friction material **18** feeding device **49** can be either of the first, second or third inlet channels **45**, **46**, **47** (not shown). Furthermore, the friction material **48** may be a solid material, a liquid or a gas, or a combination thereof.

(110) The invention is not limited to the above examples but may be varied within the scope of the appending claims. For example, the maximum pressure and thus the minimum distance **D2** in the second channel section **10** is dependent on the total feeding rate of material in the first channel section **9**, type of material, and temperature of the material when entering the second channel section **10**.

(111) FIG. **19** schematically shows a flow chart of a method for producing a profile product by use of a device according to what has been described in connection to FIGS. **1-18**, wherein the method comprises

(112) the step shown in Box **101**

(113) feeding a material to the first channel section **9** and forming the same into a master profile **36** in the first channel section **9**,

and the step shown in Box **102** feeding the material and thus master profile **36** further to the second channel section and forming the same in the second channel section **10**,

And the step in Box **103** transforming the final profile **37** into the profile product **2**.

(114) According to one example, the method further comprises the step in box **104**, stretching the final profile **37** and/or the profile product **2** for achieving the same distance in the pattern along the production direction, i.e. to achieve equal distance between elevations and/or indentations in the pattern **40** along the production direction.

(115) According to one example, the distance between indentations in the pattern on the rotating

die is less than a distance between indentations in the corresponding pattern in the production direction on the profile product, wherein the pulling and stretching device is configured to stretch the final profile and/or the profile product so that high precision in distance between features on profile can be achieved by adjustment stretching.

## Claims

1. An extrusion- or pultrusion device for forming a profile product made from a plastically deformable material and/or viscoplastic material in a production direction, said device comprising: a rotating die, extending in a radial direction and a width direction, having two opposite first and second side walls and an outer circumferential surface extending in the width direction there between, wherein the rotating die comprises a first side portion in connection to the first side wall and a second side portion in connection to the second side wall and a mid-portion extending between the first and second side portions, and a profile definition zone having a longitudinal direction coinciding with the production direction, a height direction and a width direction being perpendicular to the height direction, comprising a through channel comprising a first channel section followed by a second channel section downstream the first channel section with reference to the production direction, wherein the rotating die is rotatable about an axis extending across the production direction and arranged to allow the outer circumferential surface to, while the rotating die rotates, exert a pressure onto a surface of the material when fed through the profile definition zone, wherein the first channel section is circumferentially delimited by one or more walls and wherein the second channel section is circumferentially delimited by the circumferential surface of the rotating die and a channel portion comprising a counter-bearing opposite the rotating die and opposing first and second channel portion side walls between the rotating die and the counter-bearing, wherein the first channel section is configured to deform the material into a master profile having a maximum height at a predetermined feeding rate dependent on material and minimum cross-sectional area with a first maximum height in the first channel section, and wherein the second channel section is configured to further deform the material into a final profile having a minimum height by the rotating die being configured to apply increasing pressure on the master profile against the counter-bearing when the master profile exits the first channel section, wherein the rotating die is configured at a minimum distance between the rotating die and the counter-bearing dependent on a maximum allowable pressure applied by the rotating die at the position of that minimum distance, wherein the maximum allowable pressure corresponds to the maximum difference in height of the master profile and the final profile and dependent on pattern in the circumferential surface of the rotating die, and wherein the rotating die is configured to be in direct contact with the plastically deformable material and/or viscoplastic material that is to be formed into the profile product.

2. The device according to claim 1, wherein the one or more walls define a first cross-section at the end of the first channel section and wherein the second channel section defines a second cross-section at a position where the distance between the circumferential surface and the counter-bearing is at a minimum, and wherein the geometry of the first channel section is different from the second channel section such that the material passing through the first channel section changes form when entering the second channel section, wherein the master profile has a first cross-section area geometry corresponding to the first cross-section and wherein the final profile has a second cross-section area geometry defined by the second cross-section, wherein the first cross-section area geometry is different from the second cross-section area geometry in any given comparable position, wherein the maximum pressure and thus the minimum distance in the second channel section is dependent on a difference of cross-section area geometry of the master profile and the cross-section area geometry of the final profile.

3. The device according to claim 1, wherein the rotating die comprises a pattern comprising at least

one indentation, wherein the rotating die is configured at a maximum distance between a bottom of the indentation and the counter-bearing dependent on a minimum allowable pressure applied by the rotating die at the position of that maximum distance for achieving plastic deformation of the material in the indentation.

4. The device according to claim 2, wherein the minimum distance in the height direction between the circumferential surface and the counter-bearing in the second cross-section is less than a maximum distance in the height direction in the first cross-section.

5. The device according to claim 1, wherein the rotating die is configured, before forming the profile product, to alter form during forming of the final profile dependent on the maximum allowable pressure and/or wherein the counter-bearing is configured, before forming the profile product, to alter form during forming of the profile product dependent on the maximum allowable pressure.

6. The device according to claim 1, wherein the maximum pressure and thus the minimum distance in the second channel section is dependent on the total feeding rate of material in the first channel section, type of material, and temperature of the material when entering the second channel section.

7. The device according to claim 1, wherein the maximum allowable pressure applied by the rotating die at the position of the minimum distance is dependent on friction between the material and the counter bearing in the second channel section.

8. The device according to claim 1, wherein the cross-section area of the second channel section is configured to be sized with regard to a shrinking effect of the final profile cooling down to the profile product having a final height.

9. The device according to claim 1, wherein the pattern in the rotating die is configured to be sized with regard to a shrinking effect of the final profile cooling down to the profile product.

10. The device according to claim 1, wherein the rotating die is configured with a pattern with at least one indentation, wherein each indentation comprises a release angle dependent on the radius of the rotating die, the intended pattern in the final profile, the configuration of the counter bearing and travelling speed of the final profile.

11. The device according to claim 1, wherein the device is configured to feed a friction material between the counter-bearing and the final profile and/or configured to feed a friction material between the rotating die and the final profile.

12. The device according to claim 1, wherein the device comprises a pulling and stretching device arranged downstream the second channel section and configured to pull the material exiting the second channel section for transforming the final profile to the profile product.

13. The device according to claim 1, wherein the device comprises a pulling and stretching device, and wherein a distance between indentations in the pattern on the rotating die is less than a distance between elevations in the corresponding pattern in the production direction on the profile product, wherein the pulling and stretching device is configured to stretch the final profile and/or the profile product so that high precision in distance between features on profile can be achieved by adjustment stretching.

14. The device according to claim 1, wherein the rotating die comprises a cooling device configured to cool down the external surface of the rotating die so that the temperature of the rotating die surface is below a predetermined allowed temperature of the extruded material.

15. The device according to claim 14, wherein the rotating die is cooled on the surface so that the temperature of the rotating die surface is at least 10 degrees Celsius below a glass transition temperature or melting temperature of the material.

16. The device according to claim 14, wherein the rotating die is cooled on the surface so that the temperature of the rotating die surface is at least 50 degrees Celsius below a glass transition temperature or melting temperature of the material, enabling higher speed of extruding.

17. The device according to claim 1, wherein the material that is fed into the device to form the profile product is either one homogenous material or a mixture of two or more materials that are

blended and or layered.

18. A co-extrusion device and/or an on-extrusion device comprising the device according to claim 1, wherein the device comprises at least two inlet channels that connects directly or indirectly to the second channel section, wherein each of the at least two inlet channels is configured to feed one or more materials at a predetermined distance upstream from the second channel section or to a marriage point for the at least two inlet channels in connection to where the first channel section transitions into the second channel section.

19. A method for producing a profile product by use of the device according to claim 1, wherein the method comprises feeding a material to the first channel section and forming the same into a master profile (36) in the first channel section, feeding the material further to the second channel section and forming the same into a final profile in the second channel section, transforming the final profile into the profile product.

20. The method according to claim 19, wherein the final profile and/or the profile product is stretched for achieving a same distance in the pattern along the production direction.

21. The device according to claim 1, wherein the first and second side portions of the rotating die comprises first and second flange portions extending in a radial direction with an extension exceeding the radial extension of at least a part of the mid-portion of the rotating die, wherein the first and second flange portions are arranged to prevent movement of the material outside the rotating die.

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