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### Cooling cage assembly for a rotary electric motor

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#### Abstract

A cooling cage assembly for a rotary electric motor includes a cooling stator arrangement, and a first cylindrical part and a second cylindrical part fixedly mounted against respective opposite sides of the cooling stator arrangement. The cooling stator arrangement includes a stator and cooling tiles mounted against and around the outer circumference of the stator. Each cooling tile includes at least one channel extending from an upper side to a lower side of the cooling tile. The cooling cage assembly further includes a first plurality of fluidic bridges arranged on the upper side of the cooling tiles and a second plurality of fluidic bridges arranged on the lower side of the cooling tiles to form, together with the channel(s) of the cooling tiles, a serpentine flow path, for a cooling fluid, around the outer circumference of the stator.

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**Inventors:** Garcia-Herreros; Iván (Saint-Sulpice, CH), Chechenev; Dmitry (Neuchâtel, CH), Moreno; Loïc (Pontarlier, FR), Pourchet; Johann (La Chaux de Gilley, FR), Mittal; Manish (Neuchâtel, CH), Veronesi; Daniele (Corcelles, CH), Liu; Xinchang (Couvét, CH)

**Applicant:** ETEL S.A. (Môtiers, CH)

**Family ID:** 1000008767585

**Assignee:** ETEL S.A. (Môtiers, CH)

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*Primary Examiner:* Patel; Tulsidas C

*Assistant Examiner:* Rodriguez; Joshua Kiel M

*Attorney, Agent or Firm:* Carter, DeLuca & Farrell LLP

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## Background/Summary

### CROSS-REFERENCE TO RELATED APPLICATIONS

(1) The present application claims priority to Application No. 22153829.1, filed in the European Patent Office on Jan. 28, 2022, which is expressly incorporated herein in its entirety by reference thereto.

### FIELD OF THE INVENTION

(2) The present invention relates to a cooling cage assembly for a rotary electric motor and to a rotary electric motor including a cooling cage assembly.

### BACKGROUND INFORMATION

(3) In certain conventional rotary electric motors, a stator lamination is installed inside a motor housing or cage. The assembly of the cage around the stator lamination is achieved by an interference fit operation, in which the cage is first heated to expand its diameter and arranged around the stator lamination. A shrink fit is obtained on the stator lamination as the cage temperature reaches the ambient temperature to guarantee efficient thermal contact and provide enough friction to withstand the torque on the stator.

(4) As temperature changes during operation of the motor, the diameter of the cage slightly increases, thereby decreasing the amount of interference or contact between the stator lamination and the cage. This is acceptable for most applications since, generally, both the stator lamination and cage are made on ferrous alloys with a similar thermal expansion coefficient (CTE), and environmental temperature variations in which the motor is operating are limited as the motor is often installed in a temperature-regulated production site.

(5) However, for transportation applications, such as in the aviation industry, both the mass and temperature requirements are different than for stationary applications. Thus, the use of ferrous alloys for both the stator lamination and housing is a limiting factor.

(6) A cooling cage for preventing or at least reducing thermal expansion of both the stator lamination and cage is a conventional approach. Cooling cages are, for example, described in U.S. Patent Application Publication No. 2003/0222519, European Patent Document No. 2 680 408, and U.S. Patent Application Publication No. 2013/0342047.

(7) To avoid having two parts with a stator lamination and housing, European Patent Document No. 1 154 548 describes integrating the cooling pipes directly within the stator lamination with cast aluminum around the pipes. U.S. Pat. No. 6,954,010 describes a similar approach.

(8) German Patent Document No. 25 42 483 describes a cooling arrangement that includes external ribbed tiles fixed to the outer circumference of the stator with an adhesive. The ribbed tiles are profiled aluminum extrusions cut to the length required for the stator.

### SUMMARY

(9) Example embodiments of the present invention provide a cooling cage assembly for a rotary electric motor.

(10) According to an example embodiment of the present invention, a cooling cage assembly for a rotary electric motor includes a cooling stator arrangement and a first cylindrical part and a second cylindrical part fixedly mounted against respective opposite sides of the cooling stator arrangement. The cooling stator arrangement includes a stator and cooling tiles mounted against and around the outer circumference of the stator. Each cooling tile includes at least one channel extending from an upper side to a lower side of the cooling tile. The cooling cage assembly further includes a first plurality of fluidic bridges and a second plurality of fluidic bridges arranged respectively on the upper side and on the lower side of the cooling tiles to form, together with the channel of respective cooling tiles, a serpentine flow path for a cooling fluid around the outer circumference of the stator.

(11) According to example embodiments, each cooling tile includes a first channel and a second channel extending from the upper side to the lower side of the cooling tile. Each fluidic bridge of the first plurality of fluidic bridges is in fluid communication with the first and second channels of respective cooling tiles. Each fluidic bridge of the second plurality of fluidic bridges is in fluid communication with one channel of respective cooling tiles and with one channel of an adjacent cooling tile to form the serpentine flow path.

(12) According to example embodiments, the first plurality of fluidic bridges includes first arcuate recesses arranged on an end side of one of the first and second cylindrical parts. The first arcuate recesses are regularly spaced apart from each other over  $360^\circ$ . The second plurality of fluidic bridges includes second arcuate recesses arranged on an end side of the other of the first and second cylindrical parts. The second arcuate recesses are regularly spaced apart from each other over  $360^\circ$ . The first arcuate recesses are angularly offset with respect to the second arcuate recesses.

(13) According to example embodiments, the cooling cage assembly further includes a first ring and a second ring. The first ring includes first arcuate sealing portions arranged to cover and seal the arcuate recesses of one of the first and second cylindrical parts. Each first arcuate sealing portion includes a first aperture and a second aperture, which are part of one of the first and second plurality of fluidic bridges. The second ring includes second arcuate sealing portions arranged to cover and seal the arcuate recesses of the other of the first and second cylindrical parts. Each second arcuate sealing portion includes a first aperture and a second aperture, which are part of the other of the first and second plurality of fluidic bridges.

(14) According to example embodiments, the first ring is mounted between the first cylindrical part and the cooling stator arrangement. The first arcuate sealing portions are mounted against the lower side of respective cooling tiles with the first and second apertures aligned with the first and second channels to form, together with respective first arcuate recesses of the first cylindrical part, the first plurality of fluidic bridges. The second ring is mounted between the second cylindrical part and the cooling stator arrangement. The second arcuate sealing portions are mounted partially against the upper side of two adjacent cooling tiles with the first and second apertures aligned respectively with the first channel of one cooling tile and the second channel of an adjacent cooling tile to form, together with respective second arcuate recesses of the second cylindrical part, the second plurality of fluidic bridges.

(15) According to example embodiments, the side of the first ring mounted against the lower side of each cooling tile includes a shoulder around each aperture of respective first arcuate sealing portions. A sealing gasket is arranged and squeezed against the shoulder of each aperture and against the lower side of respective cooling tiles.

(16) According to example embodiments, the first cylindrical part includes a shoulder around each first arcuate recess. An arcuate sealing gasket is arranged and squeezed against the shoulder of each first arcuate recess and against a flat surface of the first ring.

(17) According to example embodiments, the second arcuate sealing portions of the second ring includes arcuate grooves. An arcuate sealing gasket is mounted inside each arcuate groove and squeezed against a flat surface of the second cylindrical part.

(18) According to example embodiments, the upper side of each cooling tile includes a first aperture and a second aperture opening into the first and second channels of the tile. A shoulder is arranged around the first and second apertures of each tile. A sealing gasket is arranged and squeezed against the shoulder of each aperture of respective cooling tiles and against a flat surface of the second ring.

(19) According to example embodiments, the stator includes radial extensions. An inner side of the cooling tiles is mounted against a tile receiving surface between two radial extensions.

(20) According to example embodiments, each cooling tile is made of a deformable material such as aluminum. The radius of curvature of the inner side of each cooling tile is smaller than the radius

of curvature of the corresponding tile receiving surface before assembly. The cooling tiles are screwed against the respective receiving surfaces such that the inner side of the tiles are bent to the shape of the tile receiving surfaces.

(21) According to example embodiments, the cooling tiles are assembled against a respective tile receiving surface by a sliding dovetail.

(22) According to example embodiments, each cooling tile includes a socket extending along an inner side from the upper to the lower side of the tile. Each tile receiving surface includes a tail slidably mounted into the socket of respective cooling tiles.

(23) According to example embodiments, the stator includes a cylindrical part and teeth extending from an inner side of the cylindrical part along a radial direction. The thickness of the cylindrical part between the tile receiving portion and the inner side of the cylindrical part is lower than a third of the length of the teeth along the radial direction.

(24) According to example embodiments, one of the first and second cylindrical parts further includes an inlet and an outlet port in fluid communication with the cooling stator arrangement.

(25) According to an example embodiment of the present invention, a rotary electric motor includes a cooling cage assembly as described herein.

(26) Further features and aspects of example embodiments of the present invention are described in more detail below with reference to the appended schematic Figures.

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## Description

### BRIEF DESCRIPTION OF THE DRAWINGS

(1) FIG. 1 is an exploded perspective view of a cooling cage assembly, including a cooling stator arrangement and a first cylindrical part and a second cylindrical part mounted on both sides of the cooling stator arrangement, according to an example embodiment of the present invention.

(2) FIG. 2 is another exploded perspective view of the cooling cage assembly.

(3) FIG. 3 is a perspective view of the cooling stator arrangement of the cooling cage assembly.

(4) FIG. 4 is a perspective of the cooling stator arrangement without the cooling tiles.

(5) FIG. 5 is a perspective view of a cooling tile from its inner side.

(6) FIG. 6 is a perspective view of the cooling tile from its outer side.

(7) FIG. 7 is a perspective view of sealing gaskets.

### DETAILED DESCRIPTION

(8) With reference to FIG. 1, the cooling cage assembly **10** is adapted for use in connection with a rotary electric motor and includes a cooling stator arrangement **20**, a first cylindrical part **60** fixedly mounted against a first side of the cooling stator arrangement **20**, and a second cylindrical part **70** fixedly mounted against a second opposite side of the cooling stator arrangement **20**.

(9) As illustrated in FIGS. 1 to 3, the cooling stator arrangement **20** includes a stator **22** made of a lamination stack and cooling tiles **30**, which may be, for example, formed of aluminum to reduce the overall weight of the cooling cage assembly. These cooling tiles **30** are mounted against and around an outer cylindrical wall of stator **22**. The first and second cylindrical parts **60**, **70** and the cooling tiles **30** are configured to form a serpentine flow path for a cooling fluid around the circumference of cylindrical wall of the stator **22** to efficiently cool-down the rotary electric motor when operating.

(10) Each cooling tile **30** is assembled to a corresponding tile receiving surface **29** on the outer cylindrical wall of the stator, for example, by a sliding dovetail joint. To this end, as illustrated in FIGS. 5 and 6, each cooling tile **30** includes a socket **42** extending along the middle of an inner side **40** from an upper side **34** to a lower side **38** of the cooling tile. Each tile receiving surface **29** includes a tail **25** (see, e.g., FIG. 1) slidably mounted into the socket **42** of respective cooling tiles **30**. The tail **25** includes a rail **26** as illustrated in FIG. 4, which is integral with the stator, and a rod

**28** slidably mounted inside the rail **26**. Each cooling tile **30** includes through-holes **45** receiving screws **46**, which are screwed into the corresponding rod **28** such that the cooling tiles **30** are firmly secured to the stator **22**.

(11) The radius of curvature of the inner side **40** of each cooling tile **30**, along a direction orthogonal to the lateral sides of the tile, is smaller than the radius of curvature of the corresponding tile receiving surface **29** before assembly. The cooling tiles **30** are first assembled to the stator **22** by sliding the tail **25** of respective tile receiving surfaces **29** into the socket **42** of respective cooling tiles **30**. The inner side **40** of the tiles are then bent to the shape of the tile receiving surfaces **29** upon the screwing operation to ensure an optimal thermal contact between the cooling tiles **30** and the stator **22**.

(12) With reference to FIGS. 5 and 6, the upper side **34** of each cooling tile **30** includes a first aperture **35a** and a second aperture **35b** opening into a first channel **31** and a second channel **32** arranged to extend parallelly from each other to open to a first aperture **39a** and a second aperture **39b** located on the lower side **38** of the tile.

(13) The cooling cage assembly **10** further includes a first plurality of fluidic bridges arranged to provide a fluid communication between the first and second channels **31**, **32** of each cooling tile **30** and a second plurality of fluidic bridges to provide a fluid communication between one channel **31** of each cooling tile **30** and one channel **32** of an adjacent cooling tile **30** so as to create the serpentine flow path.

(14) In the illustrated example embodiment, the first cylindrical part **60** includes an annular side mounted against one side of the stator cooling arrangement **20**. Arcuate recesses **62** are arranged and regularly spaced apart from each other over 360° on this annular side as illustrated in FIG. 1. The second cylindrical part **70** includes an annular side mounted against the opposite side of the stator cooling arrangement **20**. With reference to FIG. 2, arcuate recesses **72** are arranged and regularly spaced apart from each other over 360° on this annular side. The arcuate recesses **72** of the second cylindrical part **70** are angularly offset with respect to the arcuate recesses **62** of the first cylindrical part **60** such that the arcuate recesses of the first and second cylindrical parts **60**, **70** partly form, respectively, the first and second plurality of fluidic bridges.

(15) The cooling cage assembly **10** further includes a first ring **50a** and a second ring **50b** as illustrated in FIGS. 1 and 2. The first ring **50a** is mounted between the first cylindrical part **60** and the stator cooling arrangement **20**, whereas the second ring **50b** is mounted between the second cylindrical part **70** and the stator cooling arrangement **20**.

(16) The first ring **50a** includes arcuate sealing portions **52a** arranged to cover and seal the arcuate recesses **62** of the first cylindrical part **60**. Each arcuate sealing portion **52a** includes a first aperture **54a** and a second aperture **55a**, which are part of the corresponding fluidic bridge of the first plurality of fluidic bridges. The second ring **50b** includes arcuate sealing portions **52b** arranged to cover and seal the arcuate recesses **72** of the second cylindrical parts **70**. Each arcuate sealing portion **52b** includes a first aperture **54b** and a second aperture **55b**, which are part of the corresponding fluidic bridge of the second plurality of fluidic bridges.

(17) As illustrated in FIGS. 1 and 5, the arcuate sealing portions **52a** of the first cylindrical part **50** are mounted against the lower side **38** of respective cooling tiles **30** with the first and second apertures **54a**, **55a** of each arcuate sealing portion **52a** aligned with the first and second apertures **39a**, **39b** of the corresponding tile **30** to form, together with respective arcuate recesses **62** of the first cylindrical part **60**, the first plurality of fluidic bridges.

(18) The arcuate sealing portions **52b** of the second cylindrical part **70** are mounted partially against the upper side **34** of two adjacent cooling tiles **30** with the first and second apertures **54b**, **55b** of each arcuate sealing portion **52b** aligned respectively with the first aperture **35a** of one cooling tile and the second aperture **35b** of an adjacent cooling tile to form, together with respective second arcuate recesses **72** of the second cylindrical part **70**, the second plurality of fluidic bridges.

(19) The first and second plurality of fluidic bridges thus form, together with the stator cooling arrangement **20**, the serpentine flow path for a cooling fluid around the outer circumference of the stator.

(20) As illustrated in FIG. **1**, the second cylindrical part **70** further includes on a top side an inlet port **74** and an outlet port **76** extending and opening into adjacent arcuate recesses **72** to circulate a cooling fluid along the serpentine flow path. The inlet and outlet may also be arranged on the first cylindrical part, on the stator cooling arrangement, etc.

(21) The stator **22** includes radial extensions **24**, each including a through-hole accommodating a screw **24a** screwed into mounting lugs **66** arranged on the first cylindrical part **60**. These radial extensions provide for reducing the footprint of the stator cooling arrangement **20** since the cooling tiles **30** are mounted between these radial extensions. With reference to FIG. **4**, the stator **22** includes a cylindrical part **23a** and teeth **23b** extending from an inner side of the cylindrical part along a radial direction. To obtain a reduced footprint, the thickness **t1** of the cylindrical part **23a** between the tile receiving portion **29** and its inner side is smaller than the thickness of cylindrical part of conventional stators for rotary electric motors. The thickness **t1** may be, for example, less than a third of the length **t2** of the teeth **23b** along the radial direction.

(22) The first and second cylindrical parts **60**, **70**, the first and second rings **50a**, **50b**, and the cooling tiles **30** are adapted to accommodate sealing gaskets **80**, **82**, as illustrated in FIG. **7**, to seal the serpentine flow path. The configuration of these parts facilitates the sealing operation during the assembly of the cooling cage assembly **10**.

(23) To this end, the first cylindrical part **60** includes an arcuate shoulder **64** around each arcuate recess **62**. An arcuate sealing gasket **80** is arranged and squeezed against the shoulder **64** of each arcuate recess **62** and against a flat surface of a lower side of the first ring **50a**. An upper side of the first ring **50a**, mounted against the lower side **38** of each cooling tile **30**, includes a shoulder **56** around each aperture **54a**, **55a** of respective arcuate sealing portions **52a**. An elongated sealing gasket **82** is arranged and squeezed against the shoulder **56** of each aperture and against the lower side **38** of respective cooling tiles **30**.

(24) A shoulder **36** is arranged around the first and second apertures **35a**, **35b** arranged on the upper side **34** of each tile **30**. An elongated sealing gasket **82** is arranged and squeezed against the shoulder **36** of each aperture **35a**, **35b** of respective cooling tiles **30** and against a flat surface of a lower side of the second ring **50b**. An upper side of the second ring **50b** includes arcuate grooves. An arcuate sealing gasket **80** is mounted inside each arcuate groove and squeezed against a flat surface of the second cylindrical part **70**.

(25) The shoulders arranged on first cylindrical part **60**, on the first and second rings **50a**, **50b**, and on the cooling tiles **30** are therefore always facing upwardly. The sealing gaskets may therefore be placed to rest on their corresponding shoulder as the different parts are mounted on top of each other during the assembly process.

(26) More particularly, arcuate sealing gaskets **80** are disposed on the shoulders **64** around the arcuate recesses **62** of the first cylindrical part **60**. The lower side of the first ring **50a** is then placed on top of first cylindrical part and sealing gasket **82** are disposed on the shoulders **56** around the apertures **54a**, **54b**. The stator cooling arrangement is then placed on top of the first ring **50a**, and the screws **24a** arranged in the radial extensions **24** of the stator are then screwed into mounting lugs **66** of the first cylindrical part **60**, thereby squeezing the sealing gaskets between the different parts.

(27) Sealing gaskets **82** are then disposed on the shoulder **36** around the first and second apertures **35a**, **35b** of each cooling tile **30** before placing the second ring on top of the cooling stator arrangement **20**. Arcuate sealing gaskets **80** are disposed inside each arcuate groove on the upper side of the second ring **50b**, and the second cylindrical part **70** is placed on top of the second ring **50b**. Finally, the cooling stator arrangement **20**, the second ring **50b**, and the second cylindrical part **70** are screwed together by screws **78**, thereby squeezing the sealing gaskets therebetween.

(28) Fluidic bridges may have other configurations. For example, the fluidic bridges may be formed integrally in the first and second cylindrical parts in contact with opposite sides of the cooling stator arrangement. The first cylindrical part may include a first series of U-shaped channels regularly spaced apart from each other over 360° with apertures opening to the annular side in contact with one side of the stator cooling arrangement to provide fluid communication between the first and second channels of respective cooling tile.

(29) The second cylindrical part may include a second series of U-shaped channels which are angularly offset with respect with the first series of U-shaped channels and with apertures opening to the annular side in contact with the opposite side of the stator cooling arrangement to provide fluid communication between one channel of respective cooling tiles and with one channel of an adjacent cooling tile. In such an arrangement, the cooling cage assembly does not include the first and second rings described above.

(30) Additionally, for example, although each cooling tile of the illustrated example embodiment includes two distinct channels, the tiles may include only one channel to provide a cooling cage assembly with a simpler construction. In such a configuration, each fluidic bridge of the first and second fluidic bridges are arranged respectively on the upper side and on the lower side of the cooling tiles to extend respectively above and below two adjacent cooling tiles to bring in fluid communication the channel of respective two adjacent cooling tiles so as to form the serpentine flow path.

#### LIST OF REFERENCE NUMERALS

(31) **10** Cooling cage assembly **20** Cooling stator arrangement **22** Stator **23a** Cylindrical part **23b** Teeth **24** Radial extensions **24a** Screws **25** Tail **26** Rail **28** Rod **29** Tile receiving surface **30** Cooling tiles **31** First channel **32** Second channel **34** Upper side **35a** First aperture **35b** Second aperture **36** Shoulders **38** Lower side **39a** First aperture **39b** Second aperture **40** Inner side **42** Socket **44** Outer side **45** Through-holes **46** Screws **50a** First ring **50b** Second ring **52a** Arcuate sealing portion **52b** Arcuate sealing portion **54a** First aperture **54b** First aperture **55a** Second aperture **55b** Second aperture **56** Shoulder **60** First cylindrical part **62** Arcuate recesses **64** Shoulder **66** Mounting lugs **70** Second cylindrical part **72** Arcuate recesses **74** Inlet port **76** Outlet port **78** Screws **80** Arcuate sealing gaskets **82** Elongated sealing gaskets

#### Claims

1. A cooling cage assembly for a rotary electric motor, comprising: a cooling stator arrangement including a stator and cooling tiles mounted against and around an outer circumference of the stator, each cooling tile including an upper side, a lower side, and at least one channel extending from the upper side of the cooling tile to the lower side of the cooling tile; a first plurality of fluidic bridges arranged on the upper side of the cooling tiles; a second plurality of fluidic bridges arranged on the lower side of the cooling tiles; a first cylindrical part fixedly mounted against a first side of the cooling stator arrangement; and a second cylindrical part fixedly mounted against a second side of the cooling stator arrangement opposite the first side of the cooling stator arrangement; wherein a serpentine flow path, for a cooling fluid, around the outer circumference of the stator is formed by the first plurality of fluidic bridges, the second plurality of fluidic bridges, and the channel of the cooling tiles; and wherein each cooling tile is arranged as a stand-alone component removably assembled, individually and independently from each other cooling tile, against a corresponding tile-receiving surface of the outer circumference of the stator.
2. The cooling cage assembly according to claim 1, wherein each cooling tile includes a first channel and a second channel extending from the upper side of the cooling tile to the lower side of the cooling tile, each fluidic bridge of the first plurality of fluidic bridges being in fluid communication with the first channel and the second channels of a respective cooling tile, each fluidic bridge of the second plurality of fluidic bridges being in fluid communication with one



channel of a first respective cooling tile and with one channel of a second respective cooling tile adjacent the first respective cooling tile, to form the serpentine flow path.

3. The cooling cage assembly according to claim 1, wherein the first plurality of fluidic bridges includes first arcuate recesses arranged on an end side of a first one of the first and second cylindrical parts, the first arcuate recesses being regularly spaced apart from each other over  $360^\circ$ , the second plurality of fluidic bridges including second arcuate recesses arranged on an end side of a second one of the first and second cylindrical parts, the second arcuate recesses being regularly spaced apart from each other over  $360^\circ$ , the first arcuate recesses being angularly offset with respect to the second arcuate recesses.

4. The cooling cage assembly according to claim 3, further comprising a first ring and a second ring, the first ring including first arcuate sealing portions adapted to cover and seal the arcuate recesses of the first one of the first and second cylindrical parts, each first arcuate sealing portion including a first aperture and a second aperture that are parts of a first one of the first and second plurality of fluidic bridges, the second ring including second arcuate sealing portions adapted to cover and seal the arcuate recesses of the second one of the first and second cylindrical parts, each second arcuate sealing portion including a first aperture and a second aperture that are parts of a second one of the first and second plurality of fluidic bridges.

5. The cooling cage assembly according to claim 4, wherein the first ring is mounted between the first cylindrical part and the cooling stator arrangement, the first arcuate sealing portions being mounted against the lower side of the cooling tiles with the first and second apertures of the first arcuate sealing portion aligned with first and second channels of the cooling tile to form, together with respective first arcuate recesses of the first cylindrical part, the first plurality of fluidic bridges; and wherein the second ring is mounted between the second cylindrical part and the cooling stator arrangement, the second arcuate sealing portions being mounted partially against the upper side of two adjacent cooling tiles with the first and second apertures of the second arcuate sealing portion aligned respectively with the first channel of one cooling tile and the second channel of an adjacent cooling tile to form, together with respective second arcuate recesses of the second cylindrical part, the second plurality of fluidic bridges.

6. The cooling cage assembly according to claim 4, wherein a side of the first ring mounted against the lower side of each cooling tile includes a shoulder around each aperture of the first arcuate sealing portions, a sealing gasket being arranged and pressed against the shoulder and against the lower side of the respective cooling tile.

7. The cooling cage assembly according to claim 4, wherein the first cylindrical part includes a shoulder around each first arcuate recess, an arcuate sealing gasket being arranged and pressed against the shoulder of each first arcuate recess and against a flat surface of the first ring.

8. The cooling cage assembly according to claim 4, wherein the second arcuate sealing portions include arcuate grooves, an arcuate sealing gasket being mounted inside each arcuate groove and pressed against a flat surface of the second cylindrical part.

9. The cooling cage assembly according to claim 4, wherein the upper side of each cooling tile includes a first aperture and a second aperture opening into first and second channels of the tile, a shoulder being arranged around each first and second aperture of each tile, a sealing gasket being arranged and pressed against the shoulder of each aperture of each cooling tile and against a flat surface of the second ring.

10. The cooling cage assembly according to claim 1, wherein the stator includes radial extensions, an inner side of each cooling tile being mounted against the corresponding tile receiving surface between two radial extensions.

11. The cooling cage assembly according to claim 10, wherein each cooling tile is mounted against the corresponding tile receiving surface by a sliding dovetail.

12. The cooling cage assembly according to claim 10, wherein each cooling tile includes a socket extending along an inner side from the upper side to the lower side of the cooling tile, each tile

receiving surface including a tail slidably mounted in the socket of a respective cooling tile.

13. The cooling cage assembly according to claim 10, wherein each radial extension is arranged in a circumferential gap located between a respective adjacent pair of the cooling tiles.

14. The cooling cage assembly according to claim 1, wherein each cooling tile is formed of a deformable material, a radius of curvature of an inner side of each cooling tile being smaller than a radius of curvature of the corresponding tile receiving surface of the stator before assembly, each cooling tile being bent to a shape of the corresponding tile receiving surface by being screwed against the corresponding tile receiving surface.

15. The cooling cage assembly according to claim 14, wherein the deformable material includes an aluminum material.

16. The cooling cage assembly according to claim 1, wherein one of the first and second cylindrical parts includes an inlet port and an outlet port in fluid communication with the cooling stator arrangement.

17. The cooling cage assembly according to claim 1, wherein one of the first and second cylindrical parts includes an inlet port in fluid communication with the cooling stator arrangement.

18. The cooling cage assembly according to claim 1, wherein one of the first and second cylindrical parts includes an outlet port in fluid communication with the cooling stator arrangement.

19. The cooling cage assembly according to claim 1, further comprising an inlet port and an outlet port in fluid communication with the cooling stator arrangement.

20. The cooling cage assembly according to claim 1, wherein an inner side of each cooling tile is thermally coupled to the corresponding tile receiving surface of the stator.

21. A rotary electric motor, comprising: a cooling cage assembly including: a cooling stator arrangement including a stator and cooling tiles mounted against and around an outer circumference of the stator, each cooling tile including an upper side, a lower side, and at least one channel extending from the upper side of the cooling tile to the lower side of the cooling tile; a first plurality of fluidic bridges arranged on the upper side of the cooling tiles; a second plurality of fluidic bridges arranged on the lower side of the cooling tiles; a first cylindrical part fixedly mounted against a first side of the cooling stator arrangement; and a second cylindrical part fixedly mounted against a second side of the cooling stator arrangement opposite the first side of the cooling stator arrangement; wherein a serpentine flow path, for a cooling fluid, around the outer circumference of the stator is formed by the first plurality of fluidic bridges, the second plurality of fluidic bridges, and the channel of the cooling tiles; and wherein each cooling tile is arranged as a stand-alone component removably assembled, individually and independently from each other cooling tile, against a corresponding tile-receiving surface of the outer circumference of the stator.

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