



US 20250264253A1

(19) **United States**(12) **Patent Application Publication**
MUKAIYAMA et al.(10) **Pub. No.: US 2025/0264253 A1**(43) **Pub. Date: Aug. 21, 2025**(54) **COOLING AND HEATING DEVICE**(57) **ABSTRACT**(71) Applicant: **ADTEX Inc.**, Gunma (JP)(72) Inventors: **Hiroshi MUKAIYAMA**, Gunma (JP);
Hiroo SATO, Gunma (JP)(21) Appl. No.: **19/111,228**(22) PCT Filed: **Sep. 15, 2022**(86) PCT No.: **PCT/JP2022/034637**

§ 371 (c)(1),

(2) Date: **Mar. 12, 2025****Publication Classification**(51) **Int. Cl.**
F25B 30/02 (2006.01)(52) **U.S. Cl.**
CPC **F25B 30/02** (2013.01)

A cooling and heating apparatus includes: a refrigeration cycle circuit where a compression means, a gas cooler, a throttle means, and an evaporator are sequentially connected and a refrigerant circulates; and a circulation fluid circuit provided with a circulating pump and a heating device, in which circulation fluid that adjusts a temperature of a control target circulates, wherein the circulation fluid circuit includes: a freely openable and closable low-temperature path where the circulation fluid flows through the evaporator in such a manner that heat is exchangeable between the circulation fluid and the refrigerant, and a freely openable and closable high-temperature path where the circulation fluid flows through the gas cooler in such a manner that heat is exchangeable between the circulation fluid and the refrigerant, and the high-temperature path is provided with a high-temperature tank where the circulation fluid heated by the refrigerant in the gas cooler is stored.

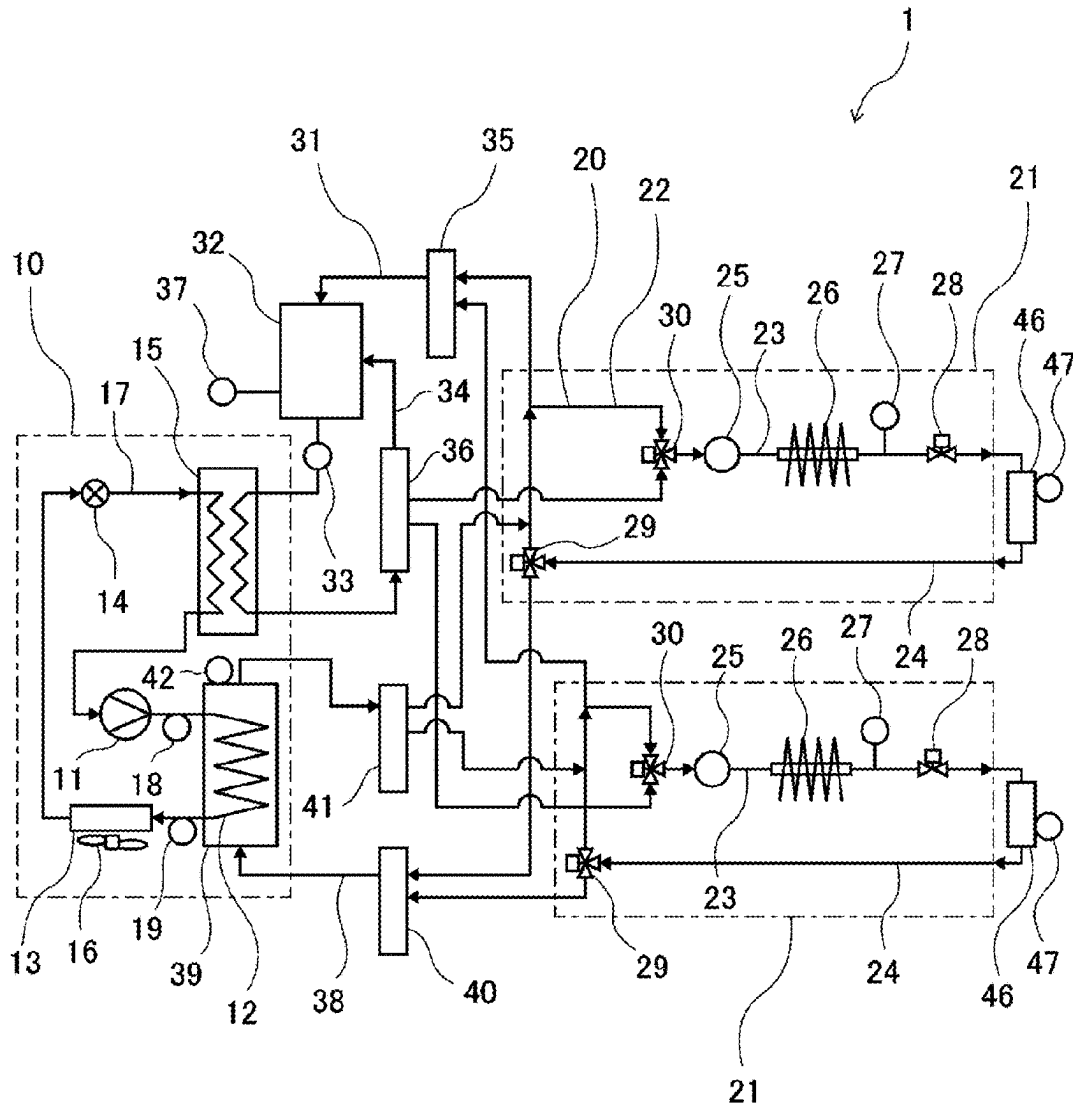


FIG. 1

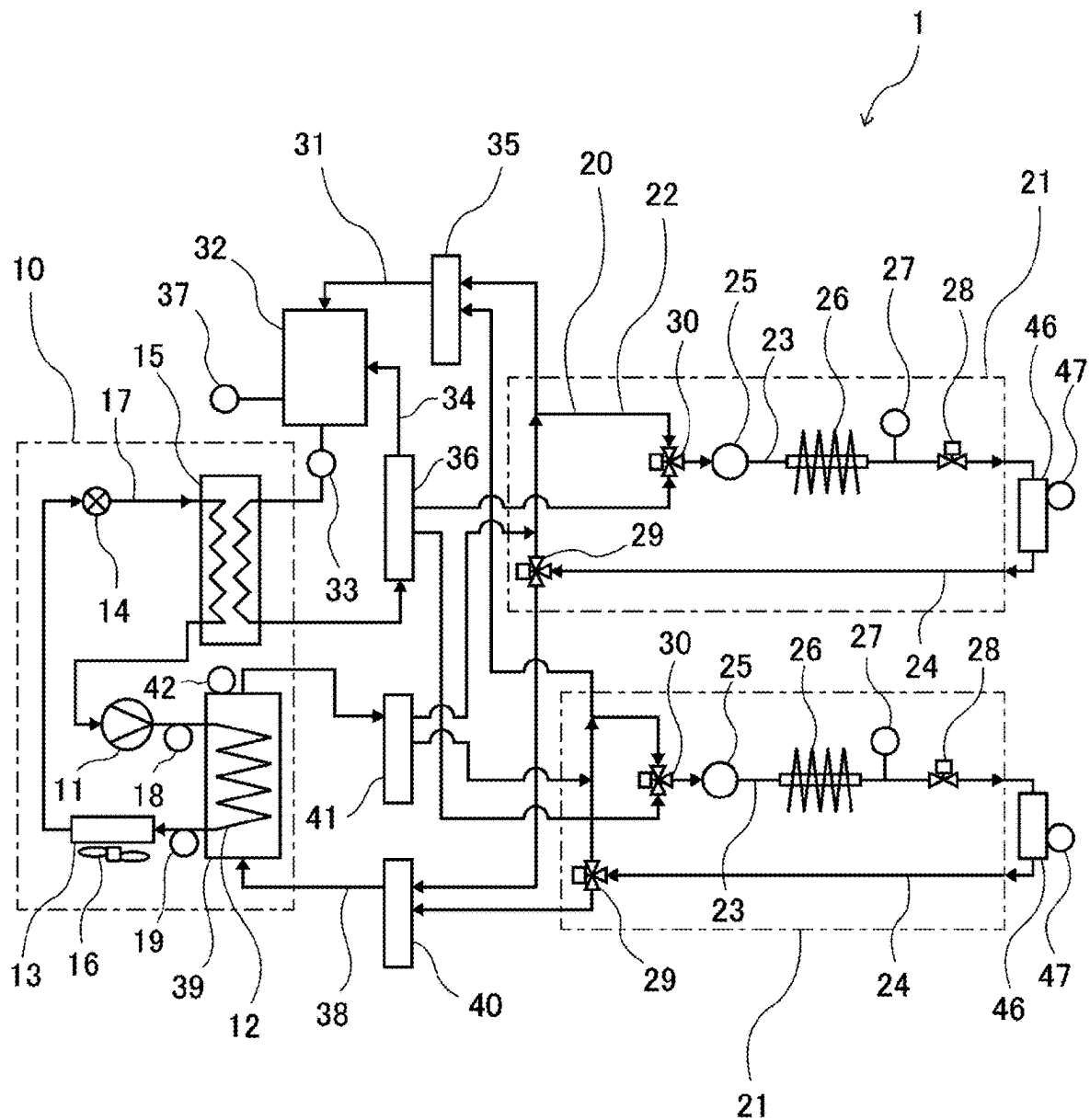


FIG. 2

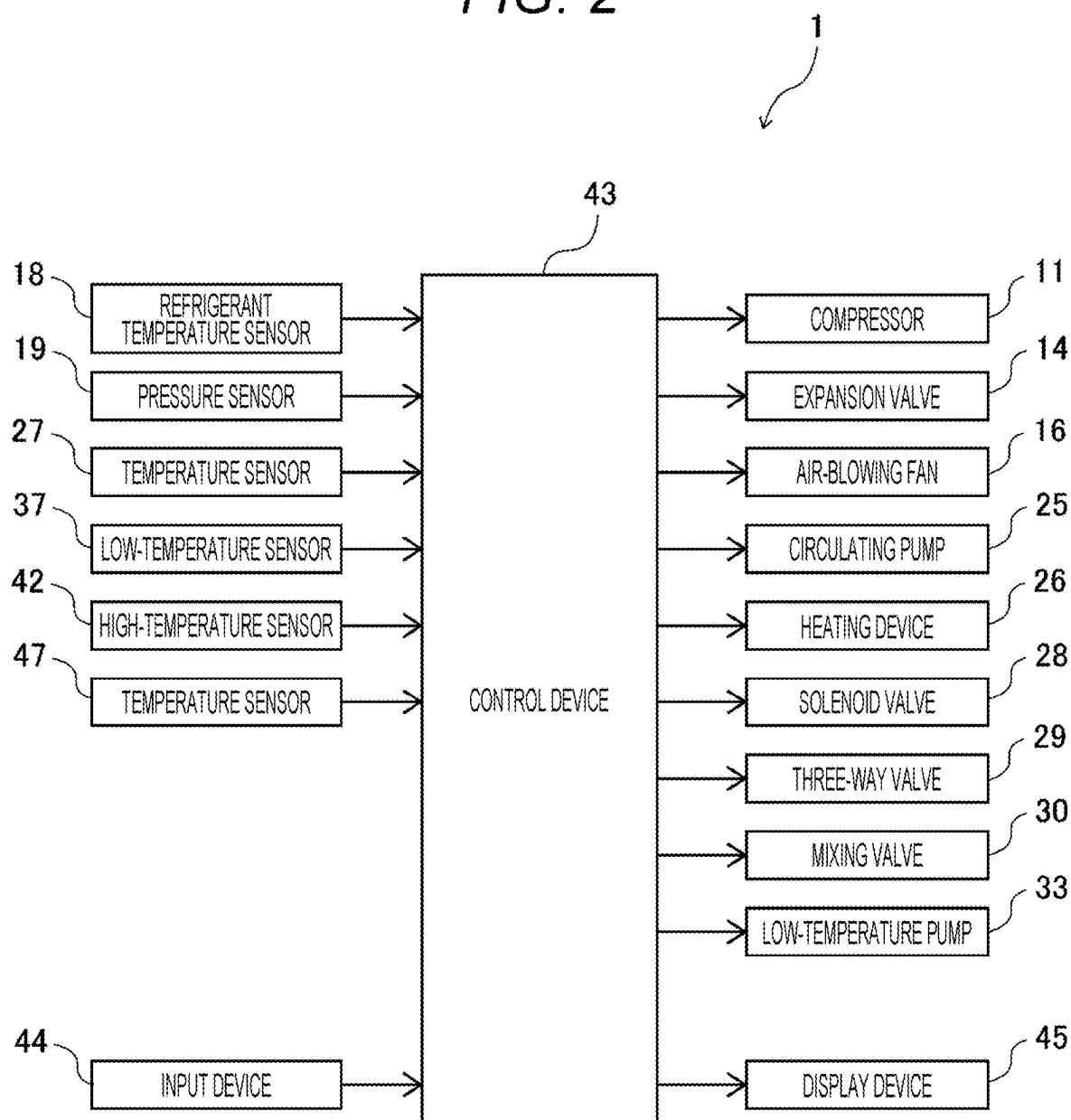


FIG. 3

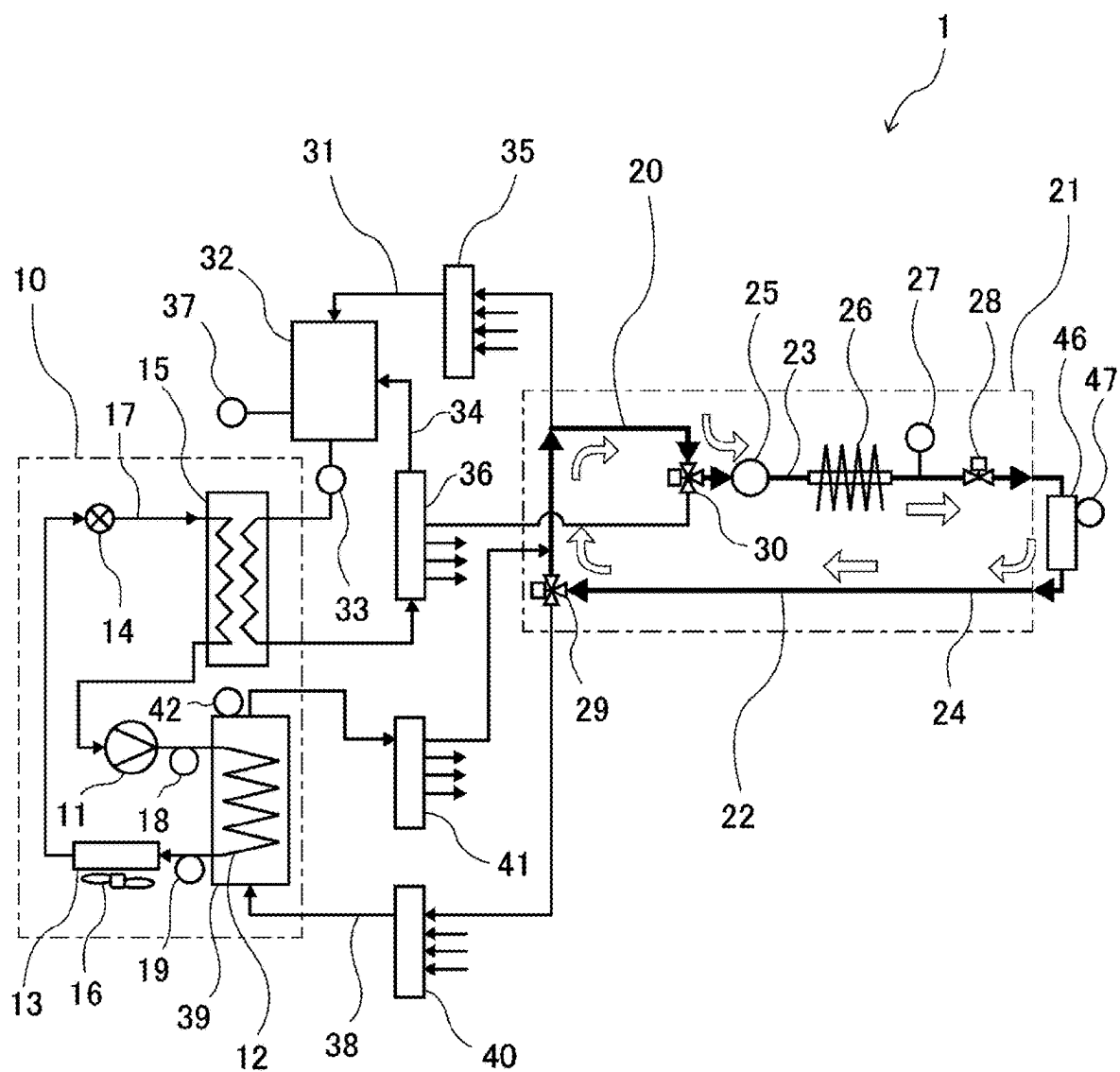


FIG. 4

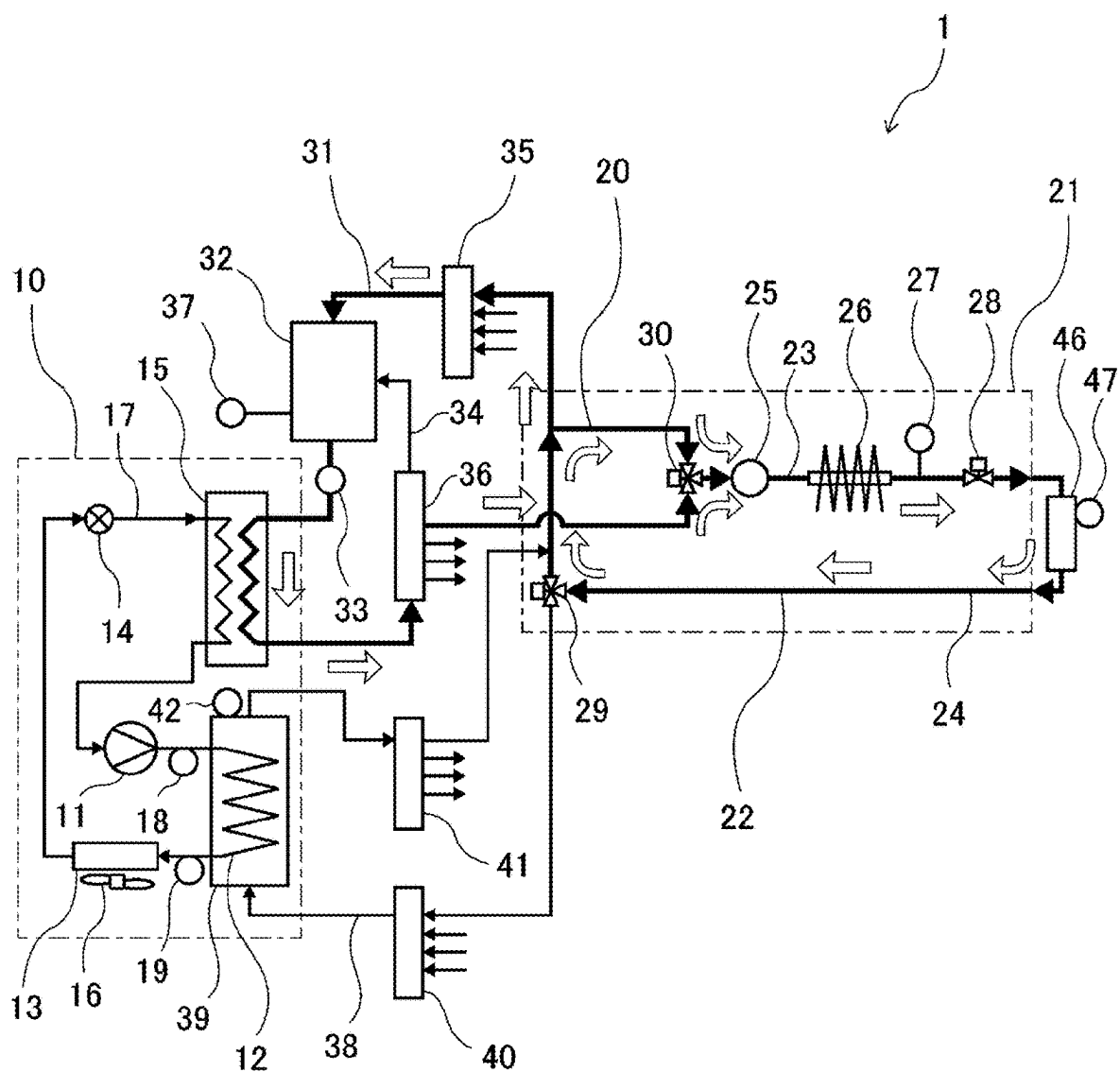


FIG. 5

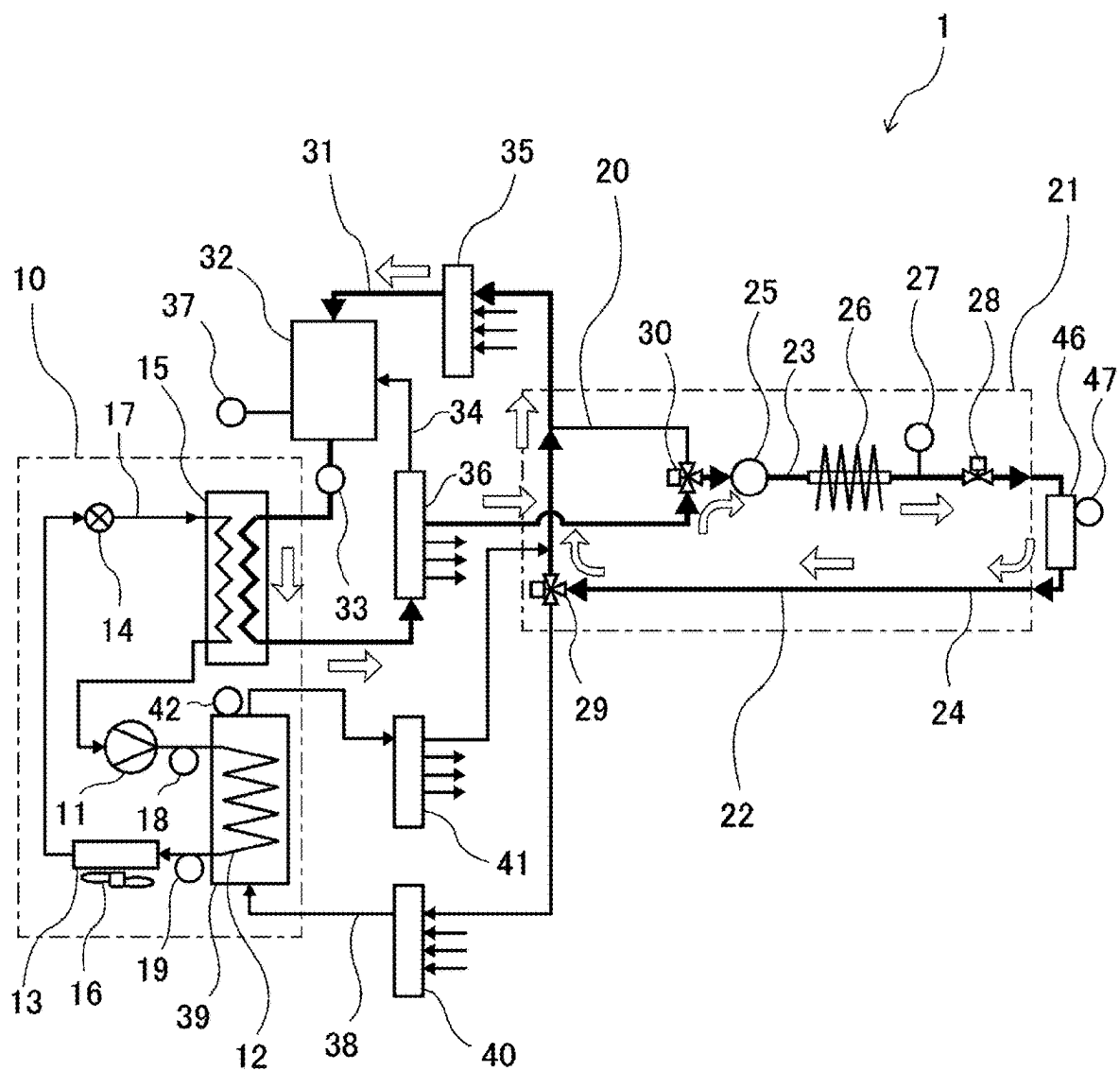
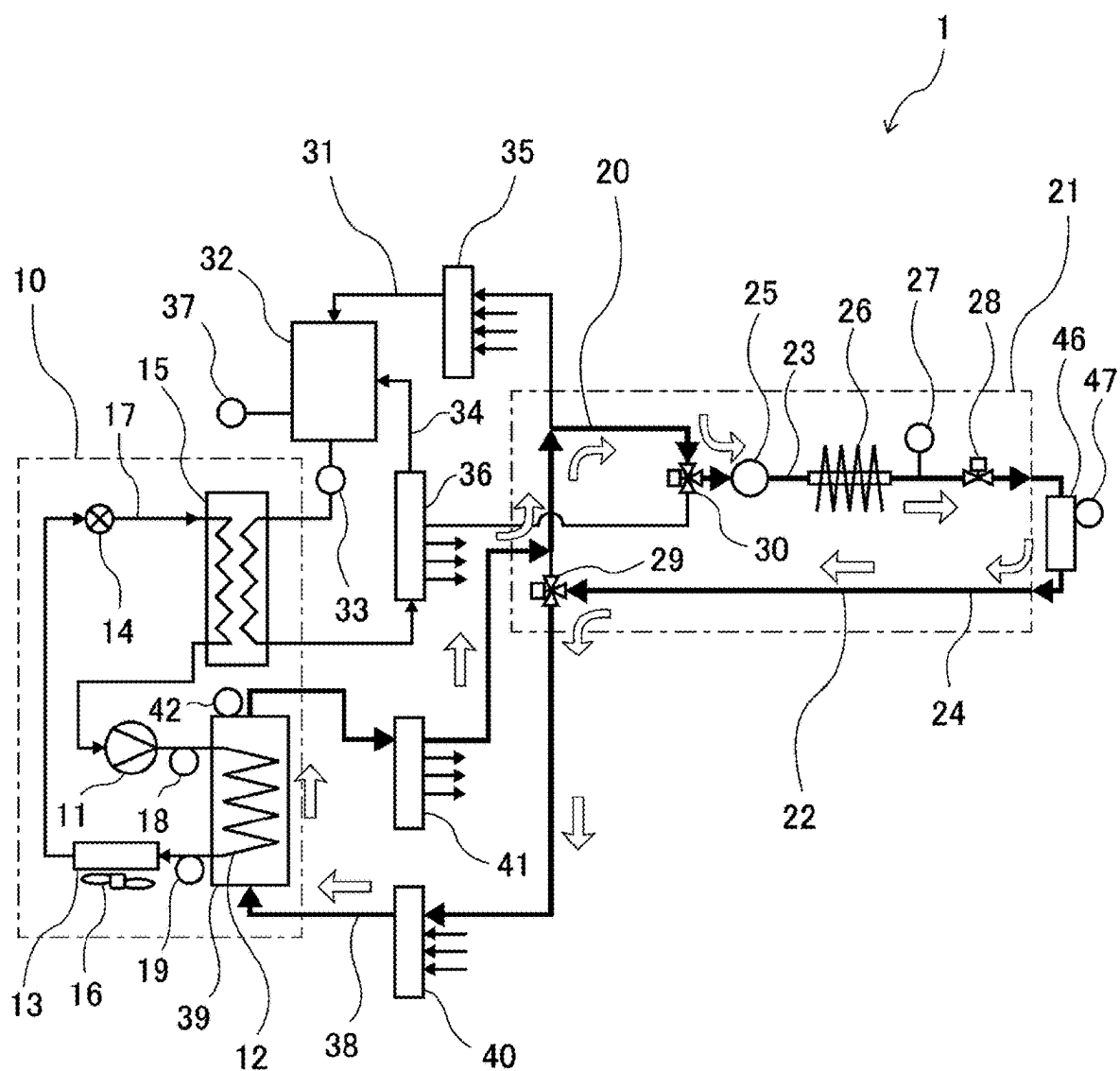


FIG. 6



COOLING AND HEATING DEVICE

TECHNICAL FIELD

[0001] The present invention relates to a cooling and heating apparatus, and particularly relates to a cooling and heating apparatus that is used to adjust, for example, various manufacturing apparatuses such as a semiconductor manufacturing apparatus and various measuring apparatuses to predetermined temperatures.

BACKGROUND ART

[0002] Generally, it is necessary for, for example, the manufacture of semiconductors to control the temperature of, for example, a manufacturing apparatus in such a manner that the temperature of, for example, a spot on a workpiece to be processed by the manufacturing apparatus or a spot to be measured reach a predetermined temperature in accordance with the manufacturing process. As an apparatus that performs such temperature control, a cooling and heating apparatus is conventionally known which includes a circulation path where a heating medium circulates, and cools or heats a control target whose temperature needs to be adjusted by use of the heating medium that circulates along the circulation path. This type of cooling and heating apparatus includes: for example, a chiller of the vapor-compression refrigeration cycle that cools a circulating heating medium; and, for example, a heating device that heats the cooled heating medium.

[0003] For example, Patent Literature 1 discloses a hybrid chiller of an area-based parameter control system that is used to control temperatures of, for example, various apparatuses such as a semiconductor manufacturing apparatus, and processes. The hybrid chiller of the area-based parameter control system disclosed in Patent Literature 1 includes: a circulation fluid circulation circuit that supplies, to a control target, circulation fluid cooled to a predetermined temperature by a refrigeration cycle; and a second circulation fluid circulation circuit that supplies, to the control target, the circulation fluid cooled to a predetermined temperature by a coolant cooled by a cooling tower. A circulation fluid supply path that feeds the circulation fluid to the control target is provided with a heater that heats the circulation fluid.

[0004] With such a configuration, the circulation fluid that is supplied to the control target is cooled, properly using a method that cools the circulation fluid by use of the refrigeration cycle and a method that cools the circulation fluid by use of the coolant of the cooling tower. The circulation fluid cooled by the refrigeration cycle or the cooling tower is heated to a predetermined temperature by the heating device such as a heater, and is supplied to the control target.

[0005] Moreover, for example, Patent Literature 2 discloses a cooling apparatus including: a first circulation system that circulates a first refrigerant in a condenser back to the condenser through a pump, a heating device, a throttle valve, and a vaporizer; and a second circulation system that includes a heat exchanger placed in the condenser, and circulates a second refrigerant that cools the first refrigerant.

[0006] The first circulation system cools a cooling target by use of the latent heat of vaporization of the first refrigerant that boils in the vaporizer. The second circulation system includes a compressor, a second condenser, an expansion valve, and the heat exchanger, and cools and

condenses the first refrigerant by use of the latent heat of vaporization of the second refrigerant in the heat exchanger provided in the condenser of the first circulation system.

[0007] Moreover, Patent Literature 2 discloses that a second heat exchanger that heats the first refrigerant by condensing the second refrigerant is provided as the heating device of the first circulation system. The second refrigerant of the second circulation system is pressurized by the compressor, is fed to the second heat exchanger, and heats the first refrigerant of the first circulation system.

CITATION LIST

Patent Literature

- [0008] Patent Literature 1: JP-A-2015-59726
- [0009] Patent Literature 2: JP-A-2022-20088

SUMMARY OF INVENTION

Problems to be Solved by Invention

[0010] However, the above cooling and heating apparatus of the known technology needs some improvements in shortening the time required to adjust temperature and encouraging an increase in efficiency in a production process of, for example, a semiconductor manufacturing apparatus and in reducing the amount of energy consumed to adjust temperature and encouraging energy savings.

[0011] Specifically, in, for example, the manufacture of semiconductors, the temperature of a control target such as a manufacturing apparatus may be changed according to, for example, the processing process or measurement process. For example, a temperature setting for the control target may need to be changed to 130° C. after a process where temperature control is performed at a temperature setting of minus 40° C. In such a case, it takes a long time for the cooling and heating apparatus of the known technology to change the temperature of the control target to a predetermined temperature setting. In this manner, the time required to change the temperature of the control target is the loss of time in the manufacturing process.

[0012] In other words, the cooling and heating apparatus of the known technology needs to heat the circulation fluid with the heating device such as an electric heater for a long time to change a temperature setting for a control target and increase the temperature. A process of heating the circulation fluid with, for example, the heating device and increasing the temperature of the control target is performed until the temperature of the control target reaches a stable set temperature. The time required to heat the circulation fluid with, for example, the heating device and increase the temperature of the control target is waiting time during which, for example, a semiconductor manufacturing apparatus cannot perform, for example, a processing process or a measurement process.

[0013] Moreover, the cooling and heating apparatus of the known technology is configured in such a manner as to cool the circulation fluid with the evaporator of the refrigeration cycle circuit and then heat the cooled circulation fluid to a predetermined temperature with the heating device such as a heater. Hence, there are problems that, for example, energy that is consumed to heat the circulation fluid, that is, the amount of electric power consumed by, for example, the heating device increases.

[0014] In contrast, Patent Literature 2 discloses that the second refrigerant of the second circulation system that cools the first refrigerant of the first circulation system by use of the latent heat of evaporation heats the first refrigerant by use of the latent heat of condensation in the second heat exchanger. In this manner, the first refrigerant corresponding to the circulation fluid to be supplied to the control target is heated by use of the latent heat of condensation of the second refrigerant being the refrigerant of the refrigeration cycle; therefore, the amount of energy consumed by, for example, the heater that is required to heat the circulation fluid can be reduced.

[0015] However, a method that heats circulation fluid by use of the latent heat of condensation of a refrigerant that is condensed by a condenser of a refrigeration cycle circuit as in the cooling apparatus disclosed in Patent Literature 2 has difficulty in heating the circulation fluid to a high temperature. Hence, even when the circulation fluid is heated by use of the latent heat of condensation of the refrigerant, if a temperature setting for a control target is high and it is necessary to heat the circulation fluid to a high temperature, much heating with a heating device such as an electric heater is required, and the amount of heating of the heating device cannot be significantly reduced.

[0016] Moreover, also in terms of the configuration that uses the condenser of the refrigeration cycle circuit to heat the circulation fluid, if the temperature setting for the control target is changed to significantly increase the temperature of the circulation fluid, it takes time to change the temperature, which leads to the loss of time before the start of, for example, a processing process or a measuring process.

[0017] The present invention has been made to solve such problems as described above. An object of the present invention is to provide a cooling and heating apparatus that can increase productivity in, for example, the manufacture of semiconductors by shortening the time required to adjust temperature, for example, upon a change in temperature setting.

[0018] Moreover, another object of the present invention is to provide a cooling and heating apparatus that can encourage energy savings by reducing the amount of energy consumed in, for example, the manufacture of semiconductors.

Solution to Problems

[0019] The cooling and heating apparatus of the present invention including a refrigeration cycle circuit where a compression means, a gas cooler, a throttle means, and an evaporator are sequentially connected and a refrigerant circulates; and a circulation fluid circuit provided with a circulating pump and a heating device, in which circulation fluid that adjusts a temperature of a control target circulates, wherein the circulation fluid circuit includes: a freely openable and closable low-temperature path where the circulation fluid flows through the evaporator in such a manner that heat is exchangeable between the circulation fluid and the refrigerant, and a freely openable and closable high-temperature path where the circulation fluid flows through the gas cooler in such a manner that heat is exchangeable between the circulation fluid and the refrigerant, and the high-temperature path is provided with a high-temperature tank where the circulation fluid heated by the refrigerant in the gas cooler is stored.

Effects of Invention

[0020] The cooling and heating apparatus includes a refrigeration cycle circuit where a compression means, a gas cooler, a throttle means, and an evaporator are sequentially connected and a refrigerant circulates; and a circulation fluid circuit provided with a circulating pump and a heating device, in which circulation fluid that adjusts a temperature of a control target circulates, the circulation fluid circuit includes: a freely openable and closable low-temperature path where the circulation fluid flows through the evaporator in such a manner that heat is exchangeable between the circulation fluid and the refrigerant, and a freely openable and closable high-temperature path where the circulation fluid flows through the gas cooler in such a manner that heat is exchangeable between the circulation fluid and the refrigerant. Such a configuration allows the cooling and heating apparatus to adjust temperature with high efficiency by supplying the circulation fluid cooled or heated by the refrigerant of the refrigeration cycle circuit to the control target such as a semiconductor manufacturing apparatus.

[0021] Specifically, if the control target needs to be cooled, the low-temperature path of the circulation fluid circuit is opened to allow the circulation fluid to flow along the low-temperature path. The circulation fluid then flows along the low-temperature path and is cooled by use of the latent heat of the refrigerant that evaporates in the evaporator of the refrigeration cycle circuit. The circulation fluid cooled in the refrigeration cycle circuit is then heated to a predetermined temperature by the heating device of the circulation fluid circuit, and is supplied at the suitable temperature to the control target in such a manner that the control target reaches a set temperature.

[0022] Moreover, if the temperature of the circulation fluid returning from the control target is low and the temperature of the circulation fluid needs to be increased sharply, the high-temperature path of the circulation fluid circuit is opened to allow the circulation fluid to flow along the high-temperature path. Consequently, the cooling and heating apparatus can heat the circulation fluid by use of heat dissipated from the refrigerant flowing through the gas cooler of the refrigeration cycle circuit. The circulation fluid heated by the gas cooler of the refrigeration cycle circuit is then heated to a predetermined temperature by the heating device of the circulation fluid circuit, and is supplied at the suitable temperature to the control target in such a manner that the control target reaches an exact set temperature. In this manner, the circulation fluid can be heated by use of the heat dissipation of the gas cooler of the refrigeration cycle circuit. Therefore, the temperature can be adjusted with high efficiency while the amount of energy to be consumed by the heating device of the circulation fluid circuit is kept low.

[0023] In this manner, the cooling and heating apparatus of the present invention can adjust temperature with low loss of exhaust heat and with high efficiency by using both cold heat and hot heat, which are generated in the refrigeration cycle circuit.

[0024] The high-temperature path is provided with a high-temperature tank where the circulation fluid heated by the refrigerant in the gas cooler is stored. Consequently, for example, if a temperature setting for the control target is changed due to a change in, for example, a processing process to significantly increase the temperature of the circulation fluid, the temperature of the circulation fluid that circulates in the circulation fluid circuit can be quickly

increased to a predetermined temperature in a short time by supplying, to the circulation fluid circuit, the high-temperature circulation fluid stored in the high-temperature tank. Hence, it is possible to significantly shorten the time required to change the temperature setting and reduce the loss of time accompanied by the change in temperature before the start of, for example, a processing process or a measurement process.

[0025] According to the cooling and heating apparatus of the present invention, the refrigerant may be carbon dioxide, and may heat the circulation fluid under supercritical pressure in the gas cooler. Consequently, it is possible to efficiently heat the circulation fluid to a high temperature.

[0026] Specifically, the cooling and heating apparatus of the present invention can heat the circulation fluid, with the gas cooler of the refrigeration cycle circuit, to a high-temperature region that is not achievable by a condenser of, for example, a chiller of a known technology using an HFC (hydrofluorocarbon)-based refrigerant, an HFO (hydrofluoroolefin)-based refrigerant, or a mixed refrigerant of them. Hence, for example, also if the temperature setting is changed to, for example, as high as 130° C. due to a change in, for example, a processing process, it is possible to increase the temperature of the circulation fluid to a high temperature in a short time. Hence, it is possible to reduce the loss of time caused by temperature adjustment and increase productivity of, for example, a semiconductor apparatus. Moreover, the amount of heating by the heating device of the circulation fluid circuit can be reduced. Therefore, it is possible to reduce the amount of energy consumed by the heating device and encourage energy savings.

[0027] According to the cooling and heating apparatus of the present invention, the refrigeration cycle circuit may include a second gas cooler that releases heat of the refrigerant to the outside, on a downstream side of the gas cooler. With such a configuration, in the refrigeration cycle circuit, heat can be dissipated from the refrigerant in the second gas cooler, and the circulation fluid in the low-temperature path can be cooled in the evaporator, even if the circulation fluid in the high-temperature tank is increased to a high temperature and there is no need to heat the circulation fluid in the high-temperature path with the refrigerant of the gas cooler. Hence, the circulation fluid is cooled by use of the refrigeration cycle to control temperature efficiently.

[0028] According to the cooling and heating apparatus of the present invention, the gas cooler may be provided in the high-temperature tank in such a manner that the refrigerant flows from up to down, and the refrigerant flowing through the gas cooler may heat the circulation fluid in the high-temperature tank. With such a configuration, the refrigerant flowing through the gas cooler can heat the circulation fluid in the high-temperature tank also in a state where the circulation fluid in the high-temperature path is not used as the circulation fluid to be supplied to the control target and is not flowing. In other words, the circulation fluid stored in the high-temperature tank can be heated to a high temperature by the gas cooler without providing, for example, a circulating pump that feeds the circulation fluid in the high-temperature path for heating with the gas cooler. Hence, when the refrigeration cycle circuit is performing operation in which the circulation fluid in the low-temperature path is cooled, exhaust heat from the gas cooler can be effectively used without circulating the circulation fluid in the high-temperature path.

[0029] According to the cooling and heating apparatus of the present invention, the low-temperature path may be provided with a low-temperature tank where the circulation fluid is stored, a low-temperature pump that feeds the circulation fluid, and a low-temperature circulation path that returns the circulation fluid to an inlet side of the low-temperature path without feeding the circulation fluid to the control target. The low-temperature tank is provided; therefore, if the temperature setting for the control target is changed due to a change in, for example, a processing process to significantly reduce the temperature of the circulation fluid, the temperature of the circulation fluid that circulates in the circulation fluid circuit can be quickly reduced to a predetermined temperature in a short time by supplying the low-temperature circulation fluid stored in the cold tank to the circulation fluid circuit. Hence, it is possible to significantly shorten the time required to change the temperature setting and reduce the loss of time accompanied by the change in temperature before the start of, for example, a processing process or a measurement process.

[0030] Moreover, the low-temperature path is provided with the low-temperature pump that feeds the circulation fluid, and the low-temperature circulation path that returns the circulation fluid from the outlet side to the inlet side of the low-temperature path. Hence, even if the circulation fluid in the low-temperature path is not used as the circulation fluid to be supplied to the control target, the circulation fluid can be cooled by the refrigerant flowing through the evaporator by circulating the circulation fluid in the low-temperature path. The refrigerant can be stored in the low-temperature tank. Moreover, even if the circulation fluid in the low-temperature path is not supplied to the control target, the circulation fluid in the high-temperature path can be heated by the refrigerant of the gas cooler by operating the refrigeration cycle circuit.

[0031] According to the cooling and heating apparatus of the present invention, the circulation fluid circuit may be provided with a three-way valve that switches between whether or not the circulation fluid returning from the control target is fed to the high-temperature path, and a mixing valve that is provided downstream of the three-way valve and mixes the circulation fluid that has passed through the low-temperature path with the circulation fluid to be supplied to the control target. With such a configuration, operation in which the circulation fluid heated by the gas cooler of the refrigeration cycle circuit is supplied to the control target and operation in which the circulation fluid cooled by the evaporator is supplied to the control target can be switched and executed by switching between the three-way valve and the mixing valve. Moreover, it is also possible to perform operation in which the mixing valve is adjusted to mix the circulation fluid cooled by the evaporator with the circulation fluid that has returned from the control target and make the temperature suitable. Furthermore, it is also possible to perform temperature adjustment operation in which the circulation fluid heated by the gas cooler and the circulation fluid cooled by the evaporator are not supplied to the control target, and only the circulation fluid that has returned from the control target is heated by the heating device, fed to the control target, and circulated. In this manner, it is possible to circulate the circulation fluid along a suitable path in accordance with the state of the control target and adjust the temperature of the control target efficiently with a little amount of energy consumed.

[0032] According to the cooling and heating apparatus of the present invention, there may be a plurality of the control targets, the circulation fluid circuit may be provided with a plurality of circuit modules connected to the low-temperature path and the high-temperature path via branch line pipes, and each of the plurality of circuit modules may include the circulating pump and the heating device, and may feed the circulation fluid to another control target. Consequently, it is possible to cool or heat control targets in, for example, a plurality of spots to be processed or measured with high efficiency by use of one refrigeration cycle circuit and adjust the control targets to their suitable temperatures.

BRIEF DESCRIPTION OF DRAWINGS

[0033] FIG. 1 is a diagram illustrating a cooling and heating apparatus according to an embodiment of the present invention.

[0034] FIG. 2 is a diagram illustrating a control system of the cooling and heating apparatus according to the embodiment of the present invention.

[0035] FIG. 3 is a diagram illustrating a circulation fluid flow path of the cooling and heating apparatus according to the embodiment of the present invention.

[0036] FIG. 4 is a diagram illustrating a circulation fluid flow path of the cooling and heating apparatus according to the embodiment of the present invention.

[0037] FIG. 5 is a diagram illustrating a circulation fluid flow path of the cooling and heating apparatus according to the embodiment of the present invention.

[0038] FIG. 6 is a diagram illustrating a circulation fluid flow path of the cooling and heating apparatus according to the embodiment of the present invention.

DESCRIPTION OF EMBODIMENTS

[0039] A cooling and heating apparatus 1 according to an embodiment of the present invention is described in detail below with appropriate reference to the drawings. Note that illustrated aspects do not limit the present invention, and are merely examples of the embodiment of the present invention.

[0040] FIG. 1 is a diagram illustrating a schematic configuration of the cooling and heating apparatus 1 according to the embodiment of the present invention. The cooling and heating apparatus 1 is an apparatus that is used to adjust a control target 46, for example, various manufacturing apparatuses such as a semiconductor manufacturing apparatus, or various measuring apparatuses used in, for example, semiconductor manufacturing processes, to a predetermined temperature according to the process (refer to FIG. 1).

[0041] The cooling and heating apparatus 1 includes a refrigeration cycle circuit 10 that configures a vapor-compression refrigeration cycle, and cools or heats circulation fluid with a refrigerant, and a circulation fluid circuit 20 that circulate the circulation fluid cooled or heated in the refrigeration cycle circuit 10 in such a manner as to feed the circulation fluid to the control target 46, and adjust the temperature of the control target 46.

[0042] Examples of the circulation fluid that circulates in the circulation fluid circuit 20 includes water. The circulation fluid is cooled or heated by the refrigerant of the refrigeration cycle circuit 10, heated to a suitable temperature by a heating device 26 of the circulation fluid circuit 20, and supplied to the control target 46 such as a semiconductor

manufacturing apparatus. Consequently, the control target 46 is controlled in such a manner as to be cooled or heated by the circulation fluid adjusted to the suitable temperature and reach a suitable temperature adequate to, for example, each manufacturing process or measurement process.

[0043] Firstly, the configuration of the refrigeration cycle circuit 10 is described in detail. The refrigeration cycle circuit 10 is formed by sequentially connecting a compressor 11 as a compression means, a gas cooler 12, a radiator 13 as a second gas cooler, an expansion valve 14 as a throttle means, and an evaporator 15 via refrigerant piping 17. The refrigeration cycle circuit 10 configures a closed circuit where the refrigerant is circulated to perform vapor-compression refrigeration cycle operation.

[0044] The compressor 11 is a compression means for compressing the refrigerant and feeding the refrigerant to the gas cooler 12. Compression devices of a rotary type, a scroll type, a reciprocating type, a screw type, and various other types can be adopted as the compressor 11.

[0045] Particularly, the compressor 11 of the rotary type is suitable to construct the cooling and heating apparatus 1 that is made compact with low cooling capacity. Moreover, the compressor 11 may be of a two-stage compression type. The adoption of the two-stage compression type as the compressor 11 is suitable to compress a carbon dioxide refrigerant whose pressure becomes high.

[0046] The gas cooler 12 is a heat exchanger that exchanges heat between the refrigerant that has been compressed to a high pressure and a high temperature by the compressor 11 and the circulation fluid of the circulation fluid circuit 20. The gas cooler 12, for example, is provided in a high-temperature tank 39 where the circulation fluid is stored and includes a plurality of tubes where the refrigerant flows although its illustration is omitted. The tubes are, for example, steel tubes.

[0047] Specifically, the tubes of the gas cooler 12 each include an inlet in an upper part thereof and an outlet in a lower part thereof to allow the refrigerant to flow from up to down, are wound into, for example, an approximately spiral shape, and are provided in the high-temperature tank 39. With such a configuration, the refrigerant that flows through the gas cooler 12 can heat the circulation fluid in the high-temperature tank 39 efficiently.

[0048] For example, even if the circulation fluid in the high-temperature tank 39 is not supplied to the control target 46, that is, even if the circulation fluid does not flow along a high-temperature path 38 of the circulation fluid circuit 20 provided with the high-temperature tank 39, the refrigerant flowing through the gas cooler 12 can heat the circulation fluid in the high-temperature tank 39.

[0049] In other words, such a configuration allows the gas cooler 12 to heat the circulation fluid stored in the high-temperature tank 39 to a high temperature without providing, for example, a circulating pump that feeds the circulation fluid to the high-temperature path 38 of the circulation fluid circuit 20 to heat the circulation fluid with the gas cooler 12.

[0050] Hence, when the refrigeration cycle circuit 10 is performing operation in which the circulation fluid is cooled by use of the latent heat of evaporation of the evaporator 15, it is possible to heat the circulation fluid in the high-temperature tank 39 to a high temperature by effectively using exhaust heat from the gas cooler 12 without circulating the circulation fluid along the high-temperature path 38.

[0051] Note that the gas cooler 12 may be provided outside the high-temperature tank 39 as long as it has a configuration that can exchange heat between the refrigerant and the circulation fluid. For example, heat exchangers of a plate type, a shell-and-tube type, a double-pipe type, and various other types may be adopted as the gas cooler 12.

[0052] The radiator 13 is a second gas cooler that releases the heat of the refrigerant to the outside, and is provided downstream of the gas cooler 12. The radiator 13 is, for example, an air-cooled heat exchanger to which air that exchanges heat with the refrigerant is delivered by an air-blowing fan 16. For example, the radiator 13 may be a fin-and-tube heat exchanger although its illustration is omitted. In other words, the radiator 13 includes a plurality of tubes such as steel tubes where the refrigerant flows, and a plurality of aluminum fins provided parallel to each other. The tubes are inserted into holes formed in the fins.

[0053] Note that the radiator 13 may be a water-cooled heat exchanger. Moreover, heat exchangers of a plate type, a shell-and-tube type, a double-pipe type, and various other types can be adopted as the radiator 13. Particularly, a heat exchanger of the plate type is preferable since the efficiency of heat exchange is high and the radiator 13 can be made compact.

[0054] The radiator 13 is provided downstream of the gas cooler 12. Therefore, the refrigerant that has dropped in temperature due to heating of the circulation fluid in the gas cooler 12 can be cooled to a lower temperature. Moreover, also if the circulation fluid in the high-temperature tank 39 is increased to a high temperature and there is no need to heat the circulation fluid with the refrigerant flowing through the gas cooler 12, the high-temperature refrigerant that has passed through the gas cooler 12 can reduce in temperature to a low temperature by heat dissipation of the radiator 13. Consequently, the cooling capacity of the refrigeration cycle circuit 10, that is, the ability to cool the circulation fluid by use of the latent heat of evaporation of the refrigerant in the evaporator 15, is provided also in a state where the high-temperature tank 39 is filled with the high-temperature circulation fluid.

[0055] The expansion valve 14 is a throttle means for decompressing the high-pressure refrigerant that has reached a low temperature after passing through the gas cooler 12 and the radiator 13. Moreover, the expansion valve 14 has a function of adjusting the flow of the refrigerant. Throttle means such as an electronic expansion valve, a thermostatic expansion valve, a capillary tube, and various other types can be adopted as the expansion valve 14. If an electronic expansion valve is adopted as the expansion valve 14, the cooling and heating of the circulation fluid in the refrigeration cycle circuit 10 can be controlled with high efficiency.

[0056] The evaporator 15 is a heat exchanger that evaporates the low-pressure liquid refrigerant and cools the circulation fluid with the latent heat of evaporation. Heat exchangers of a plate type, a double-pipe type, a tube contact type, a shell-and-tube type, and various other types may be adopted as the evaporator 15.

[0057] Particularly, a heat exchanger of the plate type is preferable since the efficiency of heat exchange is high and the evaporator 15 can be made compact. Moreover, the double-pipe type and the tube contact type are excellent in easy manufacturing and processing and easy obtainability of suitable compressive strength.

[0058] The refrigerant piping 17 downstream of the evaporator 15 is connected to the compressor 11 via an unillustrated accumulator. With the above configuration, the closed circuit of the refrigeration cycle circuit 10 is formed in which the compressor 11, the gas cooler 12, the radiator 13, the expansion valve 14, and the evaporator 15 are sequentially connected.

[0059] The refrigerant used in the refrigeration cycle circuit 10 is carbon dioxide. The carbon dioxide refrigerant heats the circulation fluid under supercritical pressure in the gas cooler 12. Consequently, the circulation fluid can be efficiently heated to a high temperature.

[0060] Specifically, the gas cooler 12 of the refrigeration cycle circuit 10 can heat the circulation fluid to a high-temperature region that is not achievable by a condenser of, for example, a chiller of a known technology using an HFC-based refrigerant, an HFO-based refrigerant, or a mixed refrigerant of them.

[0061] For example, the cooling and heating apparatus 1 can increase the temperature of the circulation fluid to a high temperature in a short time also if, for example, the temperature setting is changed to as high as 130° C. due to a change in, for example, a processing process. Hence, the cooling and heating apparatus 1 can reduce the loss of time caused by temperature adjustment and increase productivity of, for example, a semiconductor apparatus. Moreover, it is possible to reduce the amount of heating by the heating device 26 of the circulation fluid circuit 20. Therefore, it is possible to reduce the amount of energy consumed by the heating device 26 and encourage energy savings in, for example, the manufacture of semiconductors.

[0062] Moreover, the refrigeration cycle circuit 10 is provided with, for example, a refrigerant temperature sensor 18 that measures the temperature of the refrigerant, and a pressure sensor 19 that measures the pressure of the refrigerant. A control device 43 (refer to FIG. 2) controls the number of rotations of the compressor 11 and the degree of opening of the expansion valve 14 on the basis of, for example, the temperature of the refrigerant measured by the refrigerant temperature sensor 18 and the pressure of the refrigerant measured by the pressure sensor 19 in addition to the temperature setting and measured temperature information of the control target 46.

[0063] Next, the circulation fluid circuit 20 is described in detail. The circulation fluid circuit 20 configures a closed circuit where the circulation fluid that cools or heats the control target 46 circulates. Specifically, the circulation fluid circuit 20 includes: a plurality of circuit modules 21 that are connected to the control targets 46 and circulate the circulation fluid; a low-temperature path 31 that is connected to the circuit modules 21 and in which the circulation fluid flows through the evaporator 15 in such a manner as to be able to exchange heat with the refrigerant; and the high-temperature path 38 that is connected to the circuit modules 21 and in which the circulation fluid flows through the gas cooler 12 in such a manner as to be able to exchange heat with the refrigerant.

[0064] Each of the circuit modules 21 is a device that supplies the circulation fluid to the control target 46 and adjusts the temperature of the control target 46. The each of the circuit modules 21 is formed with a basic circulation path 22 that is a basic closed circuit that circulates the circulation fluid. Specifically, the each of the circuit modules 21 is formed with the basic circulation path 22 being a closed

circuit where a feed path 23 that supplies the circulation fluid to the control target 46 such as a semiconductor manufacturing apparatus, and a return path 24 that returns the circulation fluid that has cooled or heated the control target 46 are connected.

[0065] The feed path 23 of the each of the circuit modules 21 is provided with a circulating pump 25 that feeds the circulation fluid to the control target 46, the heating device 26 that heats the circulation fluid to be supplied to the control target 46 and adjusts the temperature, and a temperature sensor 27 that measure the temperature of the circulation fluid heated by the heating device 26.

[0066] The heating device 26 is, for example, an electric heater of a resistance heating type, and is, for example, a sheathed heater that covers a Nichrome wire as a heating element with a metal pipe. Moreover, the heating device 26 may be a heating means of an induction heating type, and may be, for example, an induction coil connected to an unillustrated induction heating power supply.

[0067] The temperature sensor 27 is provided to the feed path 23 downstream of the heating device 26, and measures the temperature of the circulation fluid heated by the heating device 26. The circulating pump 25, the heating device 26, and the temperature sensor 27 are connected to the control device 43. The control device 43 controls the circulating pump 25 and the heating device 26 in such a manner that the temperature of the circulation fluid measured by the temperature sensor 27 reaches a predetermined temperature. Consequently, the temperature of the control target 46 is controlled to a set temperature.

[0068] Moreover, the basic circulation path 22 of the each of the circuit modules 21 is provided with a solenoid valve 28 that opens and closes the feed path 23. Consequently, if the control target 46 connected to the circuit module 21 does not require temperature control, the flow of the circulation fluid can be stopped by closing the solenoid valve 28.

[0069] The low-temperature path 31 is a path for the refrigeration cycle circuit 10 to cool the circulation fluid. The low-temperature path 31 is connected on an inlet side thereof to a return path 24 side of the circuit module 21 and on an outlet side thereof to a feed path 23 side of the circuit module 21 in such a manner as to form a bypass path for the circulation fluid in the basic circulation path 22.

[0070] In other words, the circulation fluid that circulates along the basic circulation path 22 of the circuit module 21 can flow into the low-temperature path 31 and also flow toward the feed path 23 without flowing into the low-temperature path 31, at a branch point being the inlet of the low-temperature path 31.

[0071] A junction of the outlet of the low-temperature path 31 and the basic circulation path 22 is provided with a mixing valve 30. The mixing valve 30 is a valve that mixes the circulation fluid that has passed through the low-temperature path 31 with the circulation fluid to be supplied to the control target 46 via the feed path 23 of the circuit module 21. In other words, the mixing valve 30 can freely open and close the low-temperature path 31 and freely adjust the flow rate of the low-temperature path 31.

[0072] The circulation fluid that has returned from the control target 46 is mixed with the circulation fluid cooled by the evaporation of the refrigerant by the evaporator 15 of the refrigeration cycle circuit 10 on the basis of adjustment by the mixing valve 30. Therefore, operation that achieves a suitable temperature can be performed.

[0073] Moreover, it is also possible to perform operation that does not supply the circulation fluid cooled by the evaporator 15 to the control target 46 on the basis of adjustment by the mixing valve 30. In other words, it is also possible to perform temperature adjustment operation in which only the circulation fluid that has returned from the control target 46, or only the circulation fluid heated by the gas cooler 12, is fed to the feed path 23, heated by the heating device 26, supplied to the control target 46, and circulated.

[0074] Moreover, the low-temperature path 31 is provided with a low-temperature tank 32 where the circulation fluid is stored, a low-temperature pump 33 that feeds the circulation fluid, and a low-temperature circulation path 34 that returns the circulation fluid to the inlet side of the low-temperature path 31 without feeding the circulation fluid to the control target 46.

[0075] Specifically, for example, the low-temperature tank 32 is provided on the inlet side of the low-temperature path 31. The low-temperature pump 33 is provided downstream of the low-temperature tank 32. The evaporator 15 is provided downstream of the low-temperature pump 33. The low-temperature circulation path 34 may be provided in such a manner as to connect a branch line pipe 36 provided downstream of the evaporator 15 of the low-temperature path 31, and the low-temperature tank 32 provided on the inlet side of the low-temperature path 31.

[0076] The low-temperature tank 32 is provided with a low-temperature sensor 37 that measures the temperature of the circulation fluid in the low-temperature tank 32. The low-temperature pump 33 and the low-temperature sensor 37 are connected to the control device 43. The control device 43 may control, for example, operation of the circulating pump 25 and the low-temperature pump 33 and the adjustment of the degree of opening of the mixing valve 30 by using information on the temperature of the circulation fluid measured by the low-temperature sensor 37 to make a computation.

[0077] As described above, the low-temperature path 31 is provided with the low-temperature tank 32, the low-temperature pump 33 that feeds the circulation fluid, and the low-temperature circulation path 34 that returns the circulation fluid from the outlet side to the inlet side of the low-temperature path 31. Hence, even if the circulation fluid in the low-temperature path 31 is not used as the circulation fluid to be supplied to the control target 46, the circulation fluid in the low-temperature path 31 can be cooled by the refrigerant flowing through the evaporator 15 by circulating the circulation fluid in the low-temperature path 31.

[0078] The circulation fluid cooled by the refrigerant can be stored in the low-temperature tank 32, and the stored low-temperature circulation fluid can be supplied to the circulation fluid circuit 20 if needed. For example, if the temperature setting for the control target 46 is changed due to a change in, for example, a processing process to significantly reduce the temperature of the circulation fluid, the low-temperature circulation fluid stored in the cold tank can be supplied to the circulation fluid circuit 20.

[0079] Consequently, the temperature of the circulation fluid that circulates in the circulation fluid circuit 20 can be quickly reduced to a predetermined temperature in a short time. Hence, the time required to change the temperature setting is significantly shortened; therefore, the loss of time

accompanied by the change of the temperature before the start of, for example, a processing process or a measurement process can be reduced.

[0080] Moreover, as described above, the low-temperature path 31 is provided with the low-temperature tank 32, the low-temperature pump 33, and the low-temperature circulation path 34. Hence, even if the circulation fluid in the low-temperature path 31 is not supplied to the control target 46, it is possible to operate the refrigeration cycle circuit 10 and heat the circulation fluid in the high-temperature path 38 with the refrigerant of the gas cooler 12.

[0081] The high-temperature path 38 is a path for the refrigeration cycle circuit 10 to heat the circulation fluid. The high-temperature path 38 is connected on an inlet side thereof to the return path 24 side of the circuit module 21 and on an outlet side thereof to the feed path 23 side of the circuit module 21 in such a manner as to form a bypass path for the circulation fluid in the basic circulation path 22.

[0082] Specifically, the basic circulation path 22 of the circulation fluid circuit 20 is provided with a three-way valve 29 upstream of the branch point to the low-temperature path 31. The three-way valve 29 is a valve that switches between whether or not the circulation fluid returning from the control target 46 is fed to the high-temperature path 38. In other words, the three-way valve 29 can freely open and close the high-temperature path 38.

[0083] Specifically, the inlet of the high-temperature path 38 is connected to the three-way valve 29. The outlet of the high-temperature path 38 is connected downstream of the three-way valve 29 of the basic circulation path 22 and upstream of the branch point to the low-temperature path 31.

[0084] With such a configuration, the switching of the three-way valve 29 allows switching between operation in which the circulation fluid heated by the gas cooler 12 of the refrigeration cycle circuit 10 is supplied to the control target 46, and operation in which the circulation fluid heated by the gas cooler 12 of the refrigeration cycle circuit 10 is not supplied to the control target 46, and executing the operation.

[0085] The high-temperature path 38 is provided with the high-temperature tank 39 where the circulation fluid heated to a high temperature is stored, and a high-temperature sensor 42 that measures the temperature of the circulation fluid in the high-temperature tank 39. The gas cooler 12 of the refrigeration cycle circuit 10 is provided in the high-temperature tank 39 in such a manner that the refrigerant can heat the circulation fluid.

[0086] The high-temperature tank 39 is formed with a circulation fluid inlet in a lower part thereof, and is formed with a circulation fluid outlet in an upper part thereof. Consequently, the high-temperature circulation fluid stored in the high-temperature tank 39 can be efficiently supplied to the control target 46.

[0087] In other words, the low-temperature circulation fluid returning from the control target 46 flows into the high-temperature path 38 via the three-way valve 29, and flows into the high-temperature tank 39 through the inlet formed in the lower part of the high-temperature tank 39. The high-temperature circulation fluid stored in the high-temperature tank 39 is fed to the basic circulation path 22 through the outlet formed in the upper part of the high-temperature tank 39, and is supplied to the control target 46.

[0088] In this manner, the cooling and heating apparatus 1 includes the high-temperature tank 39, and can feed the

high-temperature circulation fluid stored in the high-temperature tank 39 to the basic circulation path 22. Hence, for example, if the temperature setting for the control target 46 is changed due to a change in, for example, a processing process to significantly increase the temperature of the circulation fluid, the temperature can be changed with high efficiency.

[0089] In other words, it is possible to supply the high-temperature circulation fluid stored in the high-temperature tank 39 to the circulation fluid circuit 20 and quickly increase the temperature of the circulation fluid that circulates in the circulation fluid circuit 20 to a predetermined temperature in a short time. Hence, the cooling and heating apparatus 1 can significantly shorten the time required to change the temperature setting and reduce the loss of time accompanied by the change in temperature before the start of, for example, a processing process or a measurement process.

[0090] Note that the control device 43 may use information on the temperature of the circulation fluid in the high-temperature tank 39, the temperature being measured by the high-temperature sensor 42, to perform a computation for opening and closing control over the three-way valve 29. Consequently, the flow in the high-temperature path 38 can be controlled according to the amount of the high-temperature circulation fluid stored in the high-temperature tank 39. Hence, if the high-temperature circulation fluid stored in the high-temperature tank 39 is less than required, it is possible to prevent the loss of time in the change of temperature, which is caused by feeding the circulation fluid that is low in temperature to the basic circulation path 22.

[0091] Moreover, the low-temperature path 31 and the high-temperature path 38 are provided with line junction pipes 35 and 40 and the branch line pipe 36 and a branch line pipe 41, which connect the plurality of circuit modules 21. Specifically, the low-temperature path 31 is provided on the inlet side with the line junction pipe 35 and on the outlet side with the branch line pipe 36. The high-temperature path 38 is provided on the inlet side with the line junction pipe 40 and on the outlet side with the branch line pipe 41.

[0092] Consequently, the plurality of circuit modules 21, for example, two to eight, or more circuit modules 21, can be connected to the low-temperature path 31 and the high-temperature path 38 via the line junction pipes 35 and 40 and the branch line pipes 36 and 41.

[0093] Each of the plurality of circuit modules 21 includes the circulating pump 25 and the heating device 26, and can circulate the circulation fluid to another control target 46. Consequently, it is possible to cool or heat the control targets 46 in, for example, a plurality of spots to be processed or measured with high efficiency by use of one refrigeration cycle circuit 10 and adjust the control targets 46 to their suitable temperatures.

[0094] FIG. 2 is a block diagram illustrating a control system of the cooling and heating apparatus 1. As illustrated in FIG. 2, the cooling and heating apparatus 1 includes the control device 43 that controls the constituent equipment. The control device 43 is a control means including a microprocessor, and executes a predetermined computation to control the temperature of the control target 46 (refer to FIG. 1).

[0095] Inputs of the control device 43 are connected to sensors such as the refrigerant temperature sensor 18 that detects the temperature of the refrigerant, the pressure sensor

19 that detects the pressure of the refrigerant, the temperature sensor 27 that detects the temperature of the circulation fluid to be supplied to the control target 46, the low-temperature sensor 37 that detects the temperature of the circulation fluid in the low-temperature path 31, the high-temperature sensor 42 that detects the temperature of the circulation fluid in the high-temperature path 38, and a temperature sensor 47 that detects the temperature of the control target 46.

[0096] Outputs of the control device 43 are connected to, for example, the compressor 11, the expansion valve 14, and the air-blowing fan 16 of the refrigeration cycle circuit 10, and the circulating pump 25, the heating device 26, the solenoid valve 28, the three-way valve 29, the mixing valve 30, and the low-temperature pump 33 of the circulation fluid circuit 20.

[0097] Moreover, the control device 43 is provided with an input device 44 that inputs a temperature setting for the control target 46 and other pieces of operation information, and a display device 45 that displays information on the temperature of each portion and other pieces of control information.

[0098] Note that the control device 43 may be connected to, for example, other unillustrated sensors, information input equipment, display devices, control target equipment, and recording devices.

[0099] The control device 43 executes predetermined computations on the basis of inputs of, for example, the refrigerant temperature sensor 18, the pressure sensor 19, the temperature sensor 27, the low-temperature sensor 37, the high-temperature sensor 42, the temperature sensor 47, and the input device 44, and controls, for example, the compressor 11, the expansion valve 14, the air-blowing fan 16, the circulating pump 25, the heating device 26, the solenoid valve 28, the three-way valve 29, the mixing valve 30, and the low-temperature pump 33.

[0100] Next, a method for adjusting temperature with the cooling and heating apparatus 1 is described in detail with reference to FIGS. 3 to 6.

[0101] FIG. 3 is a diagram illustrating a circulation fluid flow path, and illustrates an example where the circulation fluid cooled or heated in the refrigeration cycle circuit 10 is not used. Note that in FIG. 3, the path where the circulation fluid flows is indicated by thick lines, and the flow direction is indicated by arrows.

[0102] As illustrated in FIG. 3, the three-way valve 29 closes the high-temperature path 38, and the mixing valve 30 closes the low-temperature path 31. Therefore, it is also possible to not supply, to the control target 46, the circulation fluid cooled by the evaporator 15 and the circulation fluid heated by the gas cooler 12. In other words, the circulation fluid circulates along the basic circulation path 22 without flowing along the low-temperature path 31 and the high-temperature path 38. In this manner, it is also possible to perform the temperature adjustment operation where the circulation fluid flowing along the low-temperature path 31 or the high-temperature path 38 is not fed to the feed path 23, but only the circulation fluid that has returned from the control target 46 is fed directly to the feed path 23, heated by the heating device 26, fed to the control target 46, and circulated.

[0103] FIG. 4 is a diagram illustrating a circulation fluid flow path in a case where the temperature adjustment operation is performed by use of the circulation fluid cooled

in the refrigeration cycle circuit 10. In FIG. 4, the path where the circulation fluid flows is indicated by thick lines, and the flow direction is indicated by arrows. As illustrated in FIG. 4, if the control target 46 needs to be cooled, the control device 43 (refer to FIG. 2) controls the mixing valve 30, and opens the low-temperature path 31 of the circulation fluid circuit 20 to allow the circulation fluid to flow along the low-temperature path 31.

[0104] As a result, a part of the circulation fluid that has returned from the control target 46 flows along the low-temperature path 31, and is cooled by use of the latent heat of the refrigerant that evaporates in the evaporator 15 of the refrigeration cycle circuit 10. The circulation fluid cooled in the refrigeration cycle circuit 10 then merges with the circulation fluid in the basic circulation path 22 that has not flowed along the low-temperature path 31, is heated to a predetermined temperature by the heating device 26, and is supplied at the suitable temperature to the control target 46 in such a manner that the control target 46 reaches a set temperature.

[0105] FIG. 5 is a diagram illustrating another example of using the circulation fluid cooled in the refrigeration cycle circuit 10. In FIG. 5, the path where the circulation fluid flows is indicated by thick lines, and the flow direction is indicated by arrows. As illustrated in FIG. 5, the mixing valve 30 may be controlled in such a manner as to open the low-temperature path 31 100%. In other words, the circulation fluid returning from the control target 46 does not pass directly through the mixing valve 30, but all the circulation fluid returning from the control target 46 passes through the low-temperature path 31. Only the circulation fluid cooled by the refrigerant in the refrigeration cycle circuit 10 is then fed to the feed path 23 through the mixing valve 30.

[0106] With such a flow path, it is possible to feed, to the circulation fluid circuit 20, a large amount of circulation fluid cooled to a low temperature in the refrigeration cycle circuit 10 and stored in the low-temperature tank 32 and to quickly reduce the temperature of the circulation fluid to be supplied to the control target 46. Hence, it is possible to reduce the loss of time in, for example, the process of changing the temperature setting and increase productivity of, for example, a semiconductor apparatus.

[0107] FIG. 6 is a diagram illustrating a circulation fluid flow path in a case where the temperature adjustment operation is performed by use of the circulation fluid heated in the refrigeration cycle circuit 10. In FIG. 5, the path where the circulation fluid flows is indicated by thick lines, and the flow direction is indicated by arrows.

[0108] If the temperature of the circulation fluid returning from the control target 46 is low and needs to be increased sharply, the control device 43 (refer to FIG. 2) controls the three-way valve 29 to open the high-temperature path 38 of the circulation fluid circuit 20. Consequently, the circulation fluid returning from the control target 46 flows along the high-temperature path 38. The circulation fluid in the high-temperature tank 39 that has increased to a high temperature by use of the heat dissipated from the refrigerant flowing through the gas cooler 12 of the refrigeration cycle circuit 10 is then fed to the basic circulation path 22 (refer to FIG. 6).

[0109] The circulation fluid heated in the refrigeration cycle circuit 10 is then fed to the feed path 23 via the mixing valve 30, heated to a predetermined temperature by the heating device 26, and supplied at the suitable temperature

to the control target 46 in such a manner that the control target 46 reaches an exact set temperature.

[0110] In this manner, the heat dissipation of the gas cooler 12 of the refrigeration cycle circuit 10 is used to heat the circulation fluid. The circulation fluid is stored in the high-temperature tank 39. The high-temperature circulation fluid stored in the high-temperature tank 39 is supplied to the basic circulation path 22. Therefore, the temperature of the circulation fluid flowing along the basic circulation path 22 can be changed to a high temperature in a short time. Hence, it is possible to keep the amount of energy consumed by the heating device 26 of the circulation fluid circuit 20 low, and adjust temperature with high efficiency.

[0111] When the control device 43 opens the three-way valve 29, the high-temperature circulation fluid stored in the high-temperature tank 39 is fed to the basic circulation path 22. After the temperature of the circulation fluid circulating along the basic circulation path 22 increases to a predetermined temperature in a short time, the control device 43 may close the three-way valve 29 to perform normal temperature adjustment operation where the circulation fluid does not flow along the high-temperature path 38, as illustrated in FIGS. 3, 4, and 5.

[0112] In other words, as illustrated in FIG. 3, the operation that adjusts temperature may be performed by only the heating device 26 heating the circulation fluid circulating along the basic circulation path 22 without using the refrigeration cycle circuit 10. Moreover, as illustrated in FIG. 4, the operation that adjusts temperature may be performed by mixing the refrigerant flowing along the low-temperature path 31 with the circulation fluid circulating along the basic circulation path 22. Moreover, as illustrated in FIG. 5, the temperature adjustment operation may be performed in which the mixing valve 30 opens the low-temperature path 31 100%, and all the circulation fluid circulating along the basic circulation path 22 is fed to the feed path 23 via the low-temperature path 31.

[0113] In other words, as illustrated in FIG. 6, after the temperature is changed with a large temperature difference due to a change in, for example, a processing process, the temperature can be adjusted by use of cooling capacity and heating capacity as small as matching the amount of heat dissipated and the amount of heat absorbed from the control target 46, as illustrated in FIGS. 3, 4, and 5.

[0114] In this manner, the cooling and heating apparatus 1 can change a temperature setting efficiently in a short time by circulating the circulation fluid along a suitable path according to the state of the control target 46, and can adjust the temperature of the control target 46 efficiently with a little amount of energy consumed.

[0115] As described above, the cooling and heating apparatus 1 according to the embodiment can adjust the temperature of the control target 46 such as a semiconductor manufacturing apparatus with high efficiency and with little loss of exhaust heat by use of both cold heat and hot heat, which are generated in the refrigeration cycle circuit 10.

[0116] Note that the present invention is not limited to the above embodiment. The present invention can undergo various modifications and implementations within the scope that does not depart from the gist of the present invention.

LIST OF REFERENCE NUMBERS

- [0117] 1 Cooling and heating apparatus
[0118] 10 Refrigeration cycle circuit

- [0119] 11 Compressor
[0120] 12 Gas cooler
[0121] 13 Radiator
[0122] 14 Expansion valve
[0123] 15 Evaporator
[0124] 16 Air-blowing fan
[0125] 17 Refrigerant piping
[0126] 18 Refrigerant temperature sensor
[0127] 19 Pressure sensor
[0128] 20 Circulation fluid circuit
[0129] 21 Circuit module
[0130] 22 Basic circulation path
[0131] 23 Feed path
[0132] 24 Return path
[0133] 25 Circulating pump
[0134] 26 Heating device
[0135] 27 Temperature sensor
[0136] 28 Solenoid valve
[0137] 29 Three-way valve
[0138] 30 Mixing valve
[0139] 31 Low-temperature path
[0140] 32 Low-temperature tank
[0141] 33 Low-temperature pump
[0142] 34 Low-temperature circulation path
[0143] 35 Line junction pipe
[0144] 36 Branch line pipe
[0145] 37 Low-temperature sensor
[0146] 38 High-temperature path
[0147] 39 High-temperature tank
[0148] 40 Line junction pipe
[0149] 41 Branch line pipe
[0150] 42 High-temperature sensor
[0151] 43 Control device
[0152] 44 Input device
[0153] 45 Display device
[0154] 46 Control target
[0155] 47 Temperature sensor

1. A cooling and heating apparatus comprising:

- a refrigeration cycle circuit where a compression means, a gas cooler, a throttle means, and an evaporator are sequentially connected and a refrigerant circulates; and
- a circulation fluid circuit provided with a circulating pump and a heating device, in which circulation fluid that adjusts a temperature of a control target circulates, wherein

the circulation fluid circuit includes:

- a freely openable and closable low-temperature path where the circulation fluid flows through the evaporator in such a manner that heat is exchangeable between the circulation fluid and the refrigerant, and
- a freely openable and closable high-temperature path where the circulation fluid flows through the gas cooler in such a manner that heat is exchangeable between the circulation fluid and the refrigerant, and the high-temperature path is provided with a high-temperature tank where the circulation fluid heated by the refrigerant in the gas cooler is stored.

2. The cooling and heating apparatus according to claim 1, wherein the refrigerant is carbon dioxide, and heats the circulation fluid under supercritical pressure in the gas cooler.

3. The cooling and heating apparatus according to claim 1, wherein the refrigeration cycle circuit includes a second

gas cooler that releases heat of the refrigerant to the outside, on a downstream side of the gas cooler.

4. The cooling and heating apparatus according to claim 1, wherein

the gas cooler is provided in the high-temperature tank in such a manner that the refrigerant flows from up to down, and

the refrigerant flowing through the gas cooler heats the circulation fluid in the high-temperature tank.

5. The cooling and heating apparatus according to claim 1, wherein the low-temperature path is provided with a low-temperature tank where the circulation fluid is stored, a low-temperature pump that feeds the circulation fluid, and a low-temperature circulation path that returns the circulation fluid to an inlet side of the low-temperature path without feeding the circulation fluid to the control target.

6. The cooling and heating apparatus according to claim 1, wherein the circulation fluid circuit is provided with a

three-way valve that switches between whether or not the circulation fluid returning from the control target is fed to the high-temperature path, and a mixing valve that is provided downstream of the three-way valve and mixes the circulation fluid that has passed through the low-temperature path with the circulation fluid to be supplied to the control target.

7. The cooling and heating apparatus according to claim 1, wherein

there are a plurality of the control targets,

the circulation fluid circuit is provided with a plurality of circuit modules connected to the low-temperature path and the high-temperature path via branch line pipes, and

each of the plurality of circuit modules includes the circulating pump and the heating device, and feeds the circulation fluid to another control target.

* * * * *