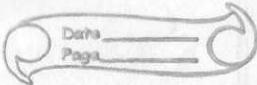


UNIT-II

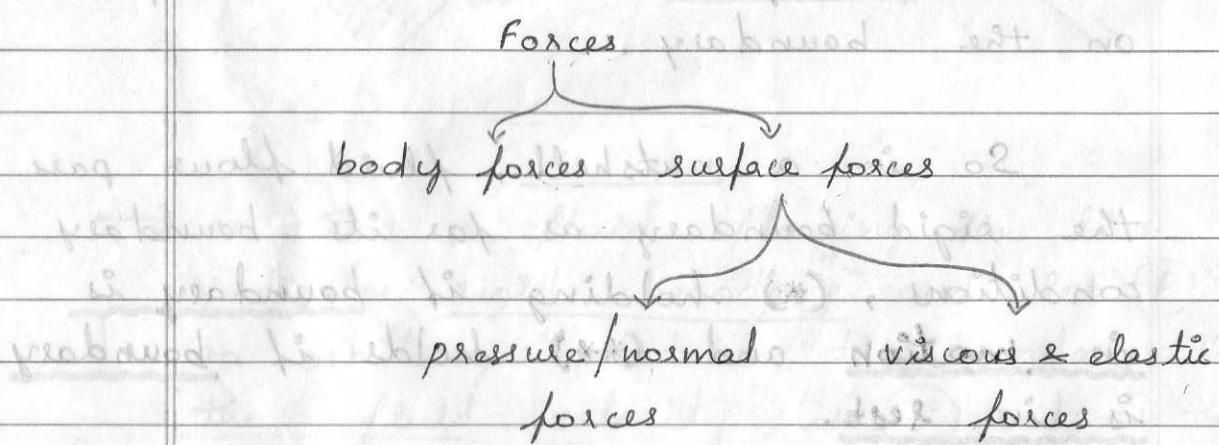


Equations of motion of a fluid

~~2.) Pressure
at a Point
is
Fluid at rest~~

The assumption is that the fluid at rest.

Forces acting on any specific mass in a fluid region can be divided into body forces (external forces (i)) and surface forces (internal in fluid) which is turns pressure forces / normal forces (ii), viscous forces and elastic forces.



~~viscous~~ Suppose the fluid flow is of invicid type then we have to consider only forces (i) & (ii).

Let P be any point within the fluid due to density there will be hydrostatic pressure.

(molecular study)

luidfluidmass
ded1) and
2) which
forces (ii),

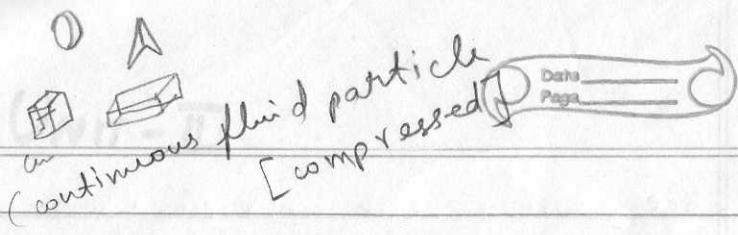
To establish the existence of hydrostatic pressure, invert a small rigid plane of area SA then either side of SA will be repeatedly bombarded with molecules of fluid moving in all directions.

As the fluid molecules impinge on one bside of the area, they transfer momentum to the area in a specified time, thereby exerting a force δF on the area.

An equal & opposite force is exerted on the fluid by this side of the area.

Suppose linear dimensions of SA are large in comparison to molecular sizes, but small when compared to macroscopic fluid length then SA is at least relative to the main body of the fluid, and motion is only on the molecular side.

Irrespective of the fact whether fluid is viscous or not, δF acts normal to SA and the mean force per unit area exerted on one side of SA will be $\frac{\delta F}{SA}$.



(Suppose we work on the molecular model of the fluid SA arbitrarily small, considered to be vanishing at P) (then as the study is in the fluid to replace molecular model by continuous fluid), we need to suppose that at P,

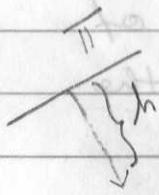
$$\lim_{SA \rightarrow 0} \left(\frac{\delta F}{\delta A} \right) \rightarrow (f) \text{ exists and is unique}$$

Thus for a fluid at rest, we have the following important results in hydrostatics.

When the fluid is at rest,

(1) Pressure at an internal point P is same at all directions and orientation of SA does not affect.

(2) Pressure at all pts at the same depth below the horizontal surface of the fluid is the same.



At point P, distance h below the free surface, the pressure is

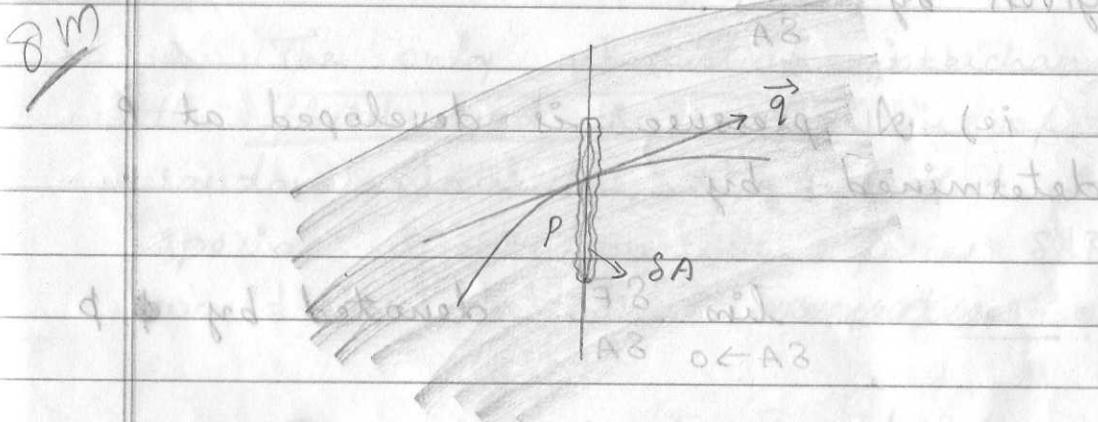
$P + \rho gh$ where P is the atmospheric pressure at the surface and ρ is the fluid density and

g is the acceleration due to gravity.

(3) If any pressure is applied to the free surface of the fluid, it is transmitted equally to all parts of the fluid.

(1), (2) & (3) are called Pascal's laws for static fluid.

Pressure at a point in a moving fluid.



Consider an inviscid fluid moving or flowing.

Suppose P is a point moving locally with the velocity \vec{v} . Insert a plane rigid area SA into the fluid at P which will also move with the velocity \vec{v} at P .

Hence relative velocity of the fluid w.r.t the plate at P at that locality is actually the velocity of the fluid with the reversed velocity of the plate (or) the fluid will be reduced.

to rest w.r.t the plane area.

Reversed velocity of fluid w.r.t
the plate at P : $\vec{q} - \vec{q} = \vec{0} \rightarrow (*)$

Case (i) :

A moment of change will take
place b/w the fluid and the plane
given by $\frac{\delta F}{SA}$.

i.e) A pressure is developed at P
determined by

$$\lim_{SA \rightarrow 0} \frac{\delta F}{SA} \text{ denoted by } \ddot{p}$$

i.e) $p = \lim_{SA \rightarrow 0} \frac{\delta F}{SA}$ exists uniquely

by ~~(f)~~ (f) ← (quote or prove based on marks)

Then the fluid pressure can be
determined at any point in the
moving fluid.

Remark:

If SA were at rest (or) moving
with a velocity \vec{v} different from
that of \vec{q} , then the local fluid
would be brought to rest as in (*).

And this requirement makes it necessary to have an additional exchange of momentum b/w the fluid and the boundary.

Case (ii) :

When SA is situated tangentially to the local fluid velocity at P.

The only momentum interchange b/w the fluid and the surface is of the random molecular type, so that for this special orientation, the forces ΣF would be the same. ~~independent of the~~

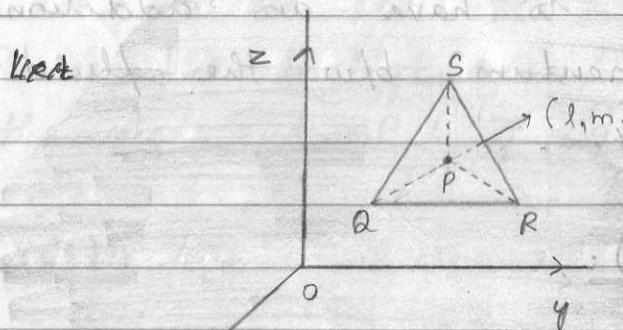
Irrespective of the fact whether SA is at rest or moving with local velocity \vec{q} or moving with some other velocity \vec{v} .

Discuss : (Question)

~~On~~ The pressure at a pt in a moving fluid is the same in all directions and is independent of the orientation of SA.

Consider a tetrahedron PARS inserted in a fluid flowing such that the edges PQ, PR, PS parallel to the coordinate axes Ox, Oy, Oz axes respectively in ~~along~~ x -increasing, y -increasing, z -increasing of a rectangular coordinate system considered

inside the fluid.



Let \vec{q} be the local velocity of the fluid at P . In the tetrahedron, the same fluid velocity \vec{q} therefore at P , the relative velocity of the fluid w.r.t. the tetrahedron is

$$\vec{q} - \vec{q} = \vec{0}$$

Ans.: The fluid is at rest w.r.t tetrahedron and hence the pressure exerted by the fluid on all the four faces of the tetrahedron may be determined to this end.

Consider the face QRS which is a base.

Let \hat{a} be the outward drawn normal to the face given by

$$\hat{a} = l\hat{i} + m\hat{j} + n\hat{k}$$

Then $|\vec{a}| = \sqrt{l^2 + m^2 + n^2}$ where (l, m, n) will be the direction cosines of \vec{a} .

Let P be the pressure exerted by the fluid on the face QRS.

Then, force exerted by the outside fluid on the face QRS with surface area δs will be given by,

$$\vec{F} = -P\delta s \hat{a} = -P\delta s (l\hat{i} + m\hat{j} + n\hat{k})$$

$$= -\frac{P}{2} (\delta s z\hat{i} + \delta s x\hat{j} + \delta s y\hat{k}) \rightarrow ①$$

$$\because \delta s = ds \cos \alpha \text{ where } \delta s \cos \alpha$$

~~is the area projected on YOZ plane~~

Suppose p_x, p_y, p_z represents the pressure exerted on faces PSR, PSQ, PQR respectively. Then the force exerted by the fluids on these faces are

$$\frac{1}{2} p_x \delta s z\hat{i}, \frac{1}{2} p_y \delta s x\hat{j}, \frac{1}{2} p_z \delta s y\hat{k} \text{ resp.}$$

i.e.) Forces exerted on face PSR = $\frac{1}{2} p_x \delta s z\hat{i} \rightarrow ②$

Forces exerted on face PSQ = $\frac{1}{2} p_y \delta s x\hat{j} \rightarrow ③$

Force exerted on face PQR = $\frac{1}{2} p_z \delta s y\hat{k} \rightarrow ④$

\therefore Total surface force exerted by the fluid on the tetrahedron is given by surface force

$$\text{Surface force } \vec{F} = ① + ② + ③ + ④$$

$$\vec{F} = -\frac{\rho}{2} (\delta_y \delta_z \hat{i} + \delta_z \delta_x \hat{j} + \delta_x \delta_y \hat{k}) + \frac{1}{2} \rho_x \delta_y \delta_z \hat{i}$$

$$+ \frac{1}{2} \rho_y \delta_z \delta_x \hat{j} + \frac{1}{2} \rho_z \delta_x \delta_y \hat{k}$$

$$= \frac{1}{2} \delta_y \delta_z (\rho_x - \rho) \hat{i} + \frac{1}{2} \delta_z \delta_x (\rho_y - \rho) \hat{j} + \frac{1}{2} \delta_x \delta_y (\rho_z - \rho) \hat{k}$$

$\hookrightarrow ⑤$

Suppose \bar{F} is the mean body force per unit mass within the tetrahedron and ρ is the mean fluid density, then

Total body force due to weight of the tetrahedron

$$= \left(\frac{1}{6} \delta_x \delta_y \delta_z \right) \rho \bar{F} \rightarrow ⑥$$

Since \bar{q} is the velocity of the fluid at P ,

$\frac{d}{dt} \bar{q} = \frac{d \bar{q}}{dt}$ is the acceleration of the fluid at P at any time t .

The force acting on fluid is given by

$$P = (\vec{F} \times m\vec{a}) = \left(\frac{1}{6} \delta_x \delta_y \delta_z \right) \rho \frac{d\vec{q}}{dt} \rightarrow (7)$$

During motion, mass of the fluid particle remains constant.

Hence the eqn of motion is given by

$$\begin{aligned} \left(\frac{1}{6} \delta_x \delta_y \delta_z \right) \rho \frac{d\vec{q}}{dt} &= \left(\frac{1}{6} \delta_x \delta_y \delta_z \right) \rho \vec{F} \\ &\quad + \frac{1}{2} (\rho_x - \rho) \delta_y \delta_z \hat{i} \\ &\quad + \frac{1}{2} (\rho_y - \rho) \delta_y \delta_z \hat{j} \\ &\quad + \frac{1}{2} (\rho_z - \rho) \delta_x \delta_y \hat{k} \end{aligned} \rightarrow (8)$$

Considering only upto 2nd order of smallness, we have to neglect the terms involving $\delta_x, \delta_y, \delta_z$.

$\therefore (8)$ yields,

$$\vec{\sigma} = \frac{1}{2} (\rho_x - \rho) \delta_y \delta_z \hat{i} + \frac{1}{2} (\rho_y - \rho) \delta_y \delta_z \hat{j} + \frac{1}{2} (\rho_z - \rho) \delta_x \delta_y \hat{k} \rightarrow (8A)$$

$$\begin{aligned} \frac{1}{2} (\rho_x - \rho) \delta_y \delta_z &= 0 ; \frac{1}{2} (\rho_y - \rho) \delta_z \delta_x = 0 ; \\ \frac{1}{2} (\rho_z - \rho) \delta_x \delta_y &= 0 \end{aligned} \rightarrow (8B)$$

s_x, s_y, s_z are small variations & not equal to 0.

Hence (8B) yields,

$$p_x - p = 0 ; p_y - p = 0 ; p_z - p = 0 \rightarrow ①$$

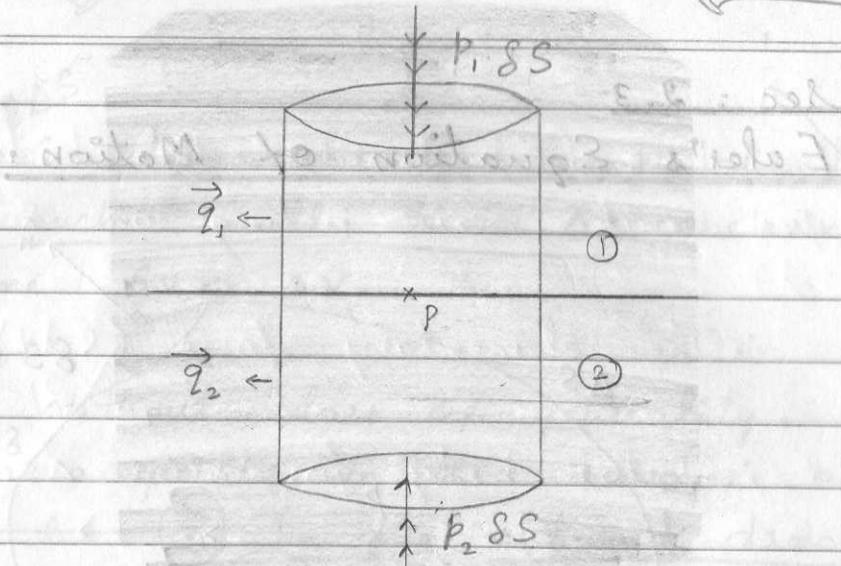
i.e.) $p_x = p_y = p_z = p$.

Hence we infer that at any point of a moving inviscid fluid, the pressure p is same in all the directions.

Thus the choice of the axes are arbitrary. We have also seen that pressure is independent of orientation of the tetrahedron and thus we infer there exists a unique pressure at each and every point in the fluid which is the same in all directions.

Conditions ^{at a point} at a boundary of two inviscid immiscible fluids:

Consider ① & ② to represent 2 inviscid immiscible fluids separated by a plane boundary and suppose P is a point on the boundary where the 2 fluids have velocities \vec{q}_1, \vec{q}_2 respectively.



Consider a small cylindrical hat-
box shaped elts of normal sections $S_1 S_2$
containing the point P projecting into
both fluids. There is no fluid transfer
across the boundary.

Generators to this surface are
normal to the surface $S_1 S_2$.

$$\therefore \rho_1 S_1 S_2 = \rho_2 S_2 S_1$$

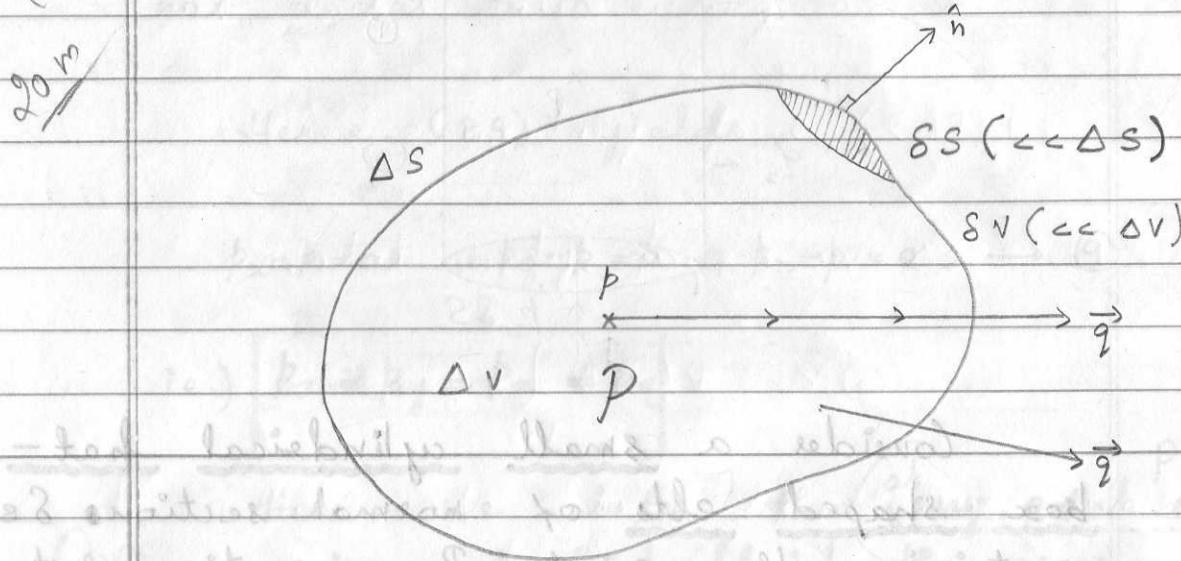
$$\text{or } \Leftrightarrow \rho_1 = \rho_2$$

(t) In particular for the case when
the fluid is in contact with the
atmosphere, the pressure at the free
surface is the same as that of the
atmosphere.

(S) If the boundary was curved, the above
conditions will get modified to account for
the effects of surface tension.

Sec : 2.3

(a) Euler's Equation of Motion:



1st part

Consider an inviscid flow.

Let ΔS be a closed region of fluid marked in the flow containing the volume ΔV .

Suppose ρ is the density of the fluid then mass of the fluid (which does not change due to conservation of mass) is $\rho \Delta V \rightarrow (f)$

Let P be a point in the fluid in the region considered.

Let the fluid element be moving with velocity \bar{q} and p be the pressure at P .

Consider an elementary closed region
 ΔS much smaller than ΔS enclosing a
volume $\Delta V \ll \Delta v$.

Without loss of generality, let p to be a point in the interior of ΔS and let \hat{n} be the outward drawn normal to ΔS at p .

\therefore Force exerted by the outside of the fluid on ΔS is given by $-p \Delta S \hat{n}$.

\therefore Total surface force exerted by the outside fluid on the entire area ΔS

$$= - \int_{\Delta S} p \hat{n} dS$$

Then By Gauss divergence theorem which states that

"Suppose V is the volume bounded by the closed surface S and \vec{A} is a vector fn of position with continuous partial derivatives, then

$$\iiint_V \nabla \cdot \vec{A} dV = \iint_S \vec{A} \cdot \hat{n} dS = \iint_S \vec{A} \cdot dS$$

where \hat{n} is a +ve normal to S

we have, $= - \int \nabla p dV \rightarrow ①$

Let \bar{F} denote the body force per unit mass acting on Δv .

\therefore The elementary force $(\rho dv) \bar{F}$.

\therefore Total body force on Δv

$$= \int_{\Delta v} \rho \bar{F} dv \rightarrow (2)$$

\downarrow Rate of change of linear momentum on the entire Δv

$$= \int_{\Delta v} \frac{d}{dt} (\rho dv \bar{q})$$

$$= \int_{\Delta v} \frac{d}{dt} \bar{q} \rho dv + \int_{\Delta v} \bar{q} \frac{d}{dt} \rho dv$$

$$= \int_{\Delta v} \frac{d}{dt} \bar{q} \rho dv \rightarrow (3)$$

$$\left[\because \int_{\Delta v} \bar{q} \frac{d}{dt} \rho dv = 0 \text{ by (f)} \right]$$

Hence eqn of motion is given by

$$\text{L.H.S } (3) = \text{L.H.S } (1) + \text{L.H.S } (2)$$

$$\Rightarrow \text{R.H.S } (3) = \text{R.H.S } (1) + \text{R.H.S } (2)$$

$$(1) \leftarrow V_b dv - =$$

force

$$\text{ie.) } \int_{\Delta V} \frac{d}{dt} \vec{q} \rho dV = - \int_{\Delta V} \nabla p dV + \int_{\Delta V} \rho \vec{F} dV$$

$$\text{ie.) } \int_{\Delta V} \left[\frac{d}{dt} \vec{q} \rho + \nabla p - \rho \vec{F} \right] dV = \vec{0}$$

Since ΔV is the volume of the arbitrary fluid element which is moving continuously with density ρ not equal to zero and pressure p locally at the point P,

$$\rho \frac{d\vec{q}}{dt} + \nabla p - \rho \vec{F} = \vec{0}$$

$$\Rightarrow \rho \frac{d\vec{q}}{dt} = \rho \vec{F} - \nabla p$$

$$\Rightarrow \frac{d\vec{q}}{dt} = \vec{F} - \frac{1}{\rho} \nabla p = \vec{0} \quad (\because \rho \neq 0) \rightarrow (I)$$

This is Euler's equation of motion.

[from relationship b/w local & particle rate of change]

We have

$$\frac{d\vec{q}}{dt} = \frac{\partial \vec{q}}{\partial t} + (\vec{q} \cdot \nabla) \vec{q} \rightarrow (4)$$

Consider,

Vector form

$$(\vec{q} \cdot \nabla) \vec{q} = \left(\vec{q} \cdot \left(\frac{\partial \vec{i}}{\partial x} \hat{i} + \frac{\partial \vec{j}}{\partial y} \hat{j} + \frac{\partial \vec{k}}{\partial z} \hat{k} \right) \right) \vec{q}$$

$$= (\vec{q} \cdot \hat{i}) \frac{\partial \vec{q}}{\partial x} + (\vec{q} \cdot \hat{j}) \frac{\partial \vec{q}}{\partial y} + (\vec{q} \cdot \hat{k}) \frac{\partial \vec{q}}{\partial z}$$

From ④,

$$\frac{d\bar{q}}{dt} = \underbrace{\frac{\partial \bar{q}}{\partial t}}_{I} + \underbrace{(\bar{q} \cdot \hat{i}) \frac{\partial \bar{q}}{\partial x}}_{II} + \underbrace{(\bar{q} \cdot \hat{j}) \frac{\partial \bar{q}}{\partial y}}_{III} + \underbrace{(\bar{q} \cdot \hat{k}) \frac{\partial \bar{q}}{\partial z}}_{IV} \rightarrow (*)$$

From the well known vector identity

$$A \times (B \times C) = (A \cdot C)B - (A \cdot B)C$$

$$(A \cdot B)C = (A \cdot C)B - A \times (B \times C)$$

we can write the 2nd, 3rd & 4th term
on RHS of (*) as

$$\begin{aligned} (\bar{q} \cdot \hat{i}) \frac{\partial \bar{q}}{\partial x} &= \left(\bar{q} \cdot \frac{\partial \bar{q}}{\partial x} \right) \hat{i} - \bar{q} \times \left(\hat{i} \times \frac{\partial \bar{q}}{\partial x} \right) \\ (\bar{q} \cdot \hat{j}) \frac{\partial \bar{q}}{\partial y} &= \left(\bar{q} \cdot \frac{\partial \bar{q}}{\partial y} \right) \hat{j} - \bar{q} \times \left(\hat{j} \times \frac{\partial \bar{q}}{\partial y} \right) \\ (\bar{q} \cdot \hat{k}) \frac{\partial \bar{q}}{\partial z} &= \left(\bar{q} \cdot \frac{\partial \bar{q}}{\partial z} \right) \hat{k} - \bar{q} \times \left(\hat{k} \times \frac{\partial \bar{q}}{\partial z} \right) \end{aligned} \quad (**)$$

$$\frac{\partial}{\partial x} (\bar{q} \cdot \bar{q}) = \frac{\partial \bar{q}}{\partial x} \cdot \bar{q} + \bar{q} \cdot \frac{\partial \bar{q}}{\partial x}$$

$$\frac{\partial}{\partial x} (\bar{q}^2) = 2 (\bar{q} \cdot \frac{\partial \bar{q}}{\partial x})$$

$$\text{Ily } \Rightarrow \frac{1}{2} \left(\frac{\partial}{\partial x} \bar{q}^2 \right) = \bar{q} \cdot \frac{\partial \bar{q}}{\partial x}$$

$$\text{IIIly, } \frac{1}{2} \left(\frac{\partial}{\partial y} \bar{q}^2 \right) = \bar{q} \cdot \frac{\partial \bar{q}}{\partial y}$$

$$\frac{1}{2} \left(\frac{\partial}{\partial z} \bar{q}^2 \right) = \bar{q} \cdot \frac{\partial \bar{q}}{\partial z}$$

Sub in (**), we get

$$(\bar{q} \cdot \hat{i}) \frac{\partial \bar{q}}{\partial x} = \frac{1}{2} \left(\frac{\partial}{\partial x} \bar{q}^2 \right) \hat{i} - \bar{q} \times \left(\hat{i} \times \frac{\partial \bar{q}}{\partial x} \right)$$

$$(\bar{q} \cdot \hat{j}) \frac{\partial \bar{q}}{\partial y} = \frac{1}{2} \left(\frac{\partial}{\partial y} \bar{q}^2 \right) \hat{j} - \bar{q} \times \left(\hat{j} \times \frac{\partial \bar{q}}{\partial y} \right)$$

$$(\bar{q} \cdot \hat{k}) \frac{\partial \bar{q}}{\partial z} = \frac{1}{2} \left(\frac{\partial}{\partial z} \bar{q}^2 \right) \hat{k} - \bar{q} \times \left(\hat{k} \times \frac{\partial \bar{q}}{\partial z} \right)$$

Adding the above relations, we get

$$\underline{(\bar{q} \cdot \hat{i}) \frac{\partial \bar{q}}{\partial x} + (\bar{q} \cdot \hat{j}) \frac{\partial \bar{q}}{\partial y} + (\bar{q} \cdot \hat{k}) \frac{\partial \bar{q}}{\partial z}}$$

$$= \frac{1}{2} \left[\frac{\partial}{\partial x} \bar{q}^2 + \frac{\partial}{\partial y} \bar{q}^2 \hat{j} + \frac{\partial}{\partial z} \bar{q}^2 \hat{k} \right]$$

$$- \bar{q} \times \left[\left(\hat{i} \times \frac{\partial \bar{q}}{\partial x} \right) + \left(\hat{j} \times \frac{\partial \bar{q}}{\partial y} \right) + \left(\hat{k} \times \frac{\partial \bar{q}}{\partial z} \right) \right]$$

$$(\bar{p} \times \nabla) \times \bar{p} - (\bar{p} \cdot \nabla) \bar{p} = \bar{q} \nabla \cdot \bar{q}$$

$$\Rightarrow (\bar{q} \cdot \nabla) \bar{q} = \frac{1}{2} [\nabla \bar{q}^2] - \bar{q} \times (\nabla \times \bar{q})$$

Sub in ④ we get,

$$\frac{d\bar{q}}{dt} = \frac{\partial \bar{q}}{\partial t} + \frac{1}{2} (\nabla \cdot \bar{q}^2) - \bar{q} \times (\nabla \times \bar{q})$$

Hence (I) ^{Euler's eqn of motion} becomes,

$$\boxed{\frac{\partial \bar{q}}{\partial t} + \nabla \left(\frac{1}{2} \bar{q}^2 \right) - \bar{q} \times (\nabla \times \bar{q}) = \bar{F} - \frac{1}{\rho} \nabla \cdot \bar{P} \rightarrow (IA)}$$

Vector form

is another form of eqn of motion
due to Euler

Tensor form of the before eqn,

$$\frac{\partial u_i}{\partial t} + u_j u_{i,j} = F_i - \frac{1}{\rho} p_{,i} \rightarrow (IB)$$

where

$\bar{x} = x_i (x_1, x_2, x_3)$ the orthonormal co-ordinate

$\bar{q} = (u_1, u_2, u_3)$ the velocity component

$\bar{F} = (F_1, F_2, F_3)$ the body force component

→ Suppose the flow is steady,

$$\frac{\partial \bar{q}}{\partial t} = 0$$

Then (IA) becomes

$$\bar{F} - \frac{1}{\rho} \nabla p = \nabla \left(\frac{1}{2} \bar{q}^2 \right) - \bar{q} \times (\nabla \times \bar{q})$$

→ Suppose the inviscid fluid flow is such that flow is irrotational.

i.e.) It is of the potential kind

$$\nabla \times \bar{q} = \bar{0} \Rightarrow \bar{q} = -\nabla \phi$$

where ϕ is the velocity potential.

Then (IA) becomes,

$$\frac{\partial \bar{q}}{\partial t} + \nabla \left(\frac{1}{2} \bar{q}^2 \right) = \bar{F} - \frac{1}{\rho} \nabla p$$

$$\bar{q} = -\nabla \phi \text{ at } \text{sub}$$

→ Suppose the inviscid flow is of the compressible type then ρ is not a constant.

Hence

$$\frac{d\bar{q}}{dt} = \bar{F} - \nabla \int_{\Delta v} \frac{dp}{\rho}$$

Equation of Motion due to Bernoulli:

Consider an inviscid flow of the potential kind where the body forces are conservative.

i.e) flow is irrotational and also that there exist a scalar velocity potential ϕ and scalar potential ψ such that

$$\bar{q} = -\nabla \phi \quad \text{and} \quad \bar{F} = -\nabla \psi$$

where \bar{F} are the body forces acting on an arbitrary fluid element under consideration to study the eqn of motion.

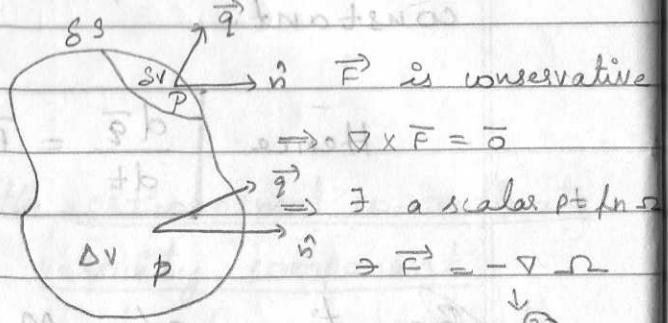
Euler's eqn of motion for an inviscid fluid is given by

$$\frac{d\bar{q}}{dt} + \frac{1}{\rho} \nabla p = \bar{F} \quad (\text{I})$$

where \bar{q} is the velocity of the fluid element at any pt P is the interior of the fluid elt and p be the pressure of the

[Here we write the 1st part of "Euler's eqn of motion" with diagram]

Motion of fluid is irrotational
 $\Rightarrow \nabla \times \vec{q} = \hat{0}$
 $\Rightarrow \exists$ vector pt ϕ



Making use of the differential operators,

$$\frac{d}{dt} = \frac{\partial}{\partial t} + (\vec{q} \cdot \nabla)$$

$$\frac{d\vec{q}}{dt} = \frac{\partial \vec{q}}{\partial t} + (\vec{q} \cdot \nabla) \vec{q}$$

$$= \frac{\partial \vec{q}}{\partial t} + \nabla \left(\frac{1}{2} \vec{q}^2 \right) - \vec{q} \times (\nabla \times \vec{q})$$

\therefore (I) becomes,

$$\frac{\partial \vec{q}}{\partial t} + \nabla \left(\frac{1}{2} \vec{q}^2 \right) - \vec{q} \times (\nabla \times \vec{q}) + \frac{1}{\rho} \nabla p = \vec{F} \rightarrow (IA)$$

$$\rightarrow \frac{\partial}{\partial t} (-\nabla \phi) + \frac{1}{2} \nabla (-\nabla \phi)^2 - (-\nabla \phi) \times (\nabla \times -\nabla \phi) + \frac{1}{\rho} \nabla p$$

$$(II) - \vec{F} = \frac{1}{\rho} \nabla p = -\nabla \Omega$$

$$\rightarrow \frac{\partial}{\partial t} (\nabla \phi) - \frac{1}{2} \nabla (+\nabla \phi)^2 + \underbrace{(-\nabla \phi) \times (\nabla \times -\nabla \phi)}_{= \nabla \Omega} - \frac{1}{\rho} \nabla p$$

$$\nabla \left(\frac{\partial \phi}{\partial t} \right) - \frac{1}{2} \nabla (\nabla \phi)^2 = \frac{1}{\rho} \nabla p + \nabla \Omega \rightarrow (IB)$$

"Euler's

Suppose \vec{x} denotes the position vector of the fluid particle at P at time t and $d\vec{x}$ representing instantaneous displacements made in the position of the particle at time t and scalar multiplying egn (IB) by $d\vec{x}$

$$d\vec{x} \cdot \nabla \left(\frac{\partial \phi}{\partial t} \right) - \frac{1}{2} d\vec{x} \cdot \nabla (\nabla \phi)^2 = d\vec{x} \cdot \frac{1}{\rho} \nabla p + d\vec{x} \cdot \nabla \underline{\underline{\sigma}} \quad (IC)$$

Consider, $\textcircled{1} \leftarrow \textcircled{2} \rightarrow \textcircled{3}$

$$d\vec{x} \cdot \Delta \underline{\underline{\sigma}} = (dx\hat{i} + dy\hat{j} + dz\hat{k}) \cdot \left(\frac{\partial}{\partial x} \underline{\underline{\sigma}}\hat{i} + \frac{\partial}{\partial y} \underline{\underline{\sigma}}\hat{j} + \frac{\partial}{\partial z} \underline{\underline{\sigma}}\hat{k} \right)$$

$$(CI) \leftarrow \quad = \frac{\partial}{\partial x} \underline{\underline{\sigma}}_{xx} dx + \frac{\partial}{\partial y} \underline{\underline{\sigma}}_{xy} dy + \frac{\partial}{\partial z} \underline{\underline{\sigma}}_{xz} dz$$

$$d\vec{x} \cdot \Delta \underline{\underline{\sigma}} = d\underline{\underline{\sigma}} \rightarrow \textcircled{3}$$

Consider,

$$d\vec{x} \cdot \nabla \left(\frac{\partial \phi}{\partial t} \right) = (dx\hat{i} + dy\hat{j} + dz\hat{k}) \left(\frac{\partial}{\partial x} \hat{i} + \frac{\partial}{\partial y} \hat{j} + \frac{\partial}{\partial z} \hat{k} \right) \left(\frac{\partial \phi}{\partial t} \right)$$

$$= \frac{\partial}{\partial x} \left(\frac{\partial \phi}{\partial t} \right) dx + \frac{\partial}{\partial y} \left(\frac{\partial \phi}{\partial t} \right) dy + \frac{\partial}{\partial z} \left(\frac{\partial \phi}{\partial t} \right) dz$$

$$d\vec{x} \cdot \nabla \left(\frac{\partial \phi}{\partial t} \right) = d \left(\frac{\partial \phi}{\partial t} \right) \rightarrow \textcircled{4}$$

Consider,

$$d\vec{x} \cdot \nabla (\nabla \phi)^2 = (dx\hat{i} + dy\hat{j} + dz\hat{k}) \left(\frac{\partial}{\partial x} \hat{i} + \frac{\partial}{\partial y} \hat{j} + \frac{\partial}{\partial z} \hat{k} \right) (\nabla \phi)^2$$

$$= \frac{\partial}{\partial x} (\nabla \phi)^2 dx + \frac{\partial}{\partial y} (\nabla \phi)^2 dy + \frac{\partial}{\partial z} (\nabla \phi)^2 dz$$

$$d\vec{r} \cdot \nabla (\nabla \phi)^2 = d(\nabla \phi)^2 \rightarrow \textcircled{5}$$

Consider,

$$d\vec{r} \cdot \left(\frac{1}{\rho} \nabla p \right) = (dx\hat{i} + dy\hat{j} + dz\hat{k}) \cdot \frac{1}{\rho} \left(\frac{\partial}{\partial x} \hat{i} + \frac{\partial}{\partial y} \hat{j} + \frac{\partial}{\partial z} \hat{k} \right) p$$

$$= \frac{1}{\rho} \left(\frac{\partial}{\partial x} (p) dx + \frac{\partial}{\partial y} (p) dy + \frac{\partial}{\partial z} (p) dz \right).$$

$$d\vec{r} \cdot \left(\frac{1}{\rho} \nabla p \right) = \frac{1}{\rho} dp \rightarrow \textcircled{6}$$

(I c) becomes, (using $\textcircled{3}, \textcircled{4}, \textcircled{5}$ & $\textcircled{6}$)

$$d\left(\frac{\partial \phi}{\partial t}\right) - \frac{1}{2} d(\nabla \phi)^2 = \frac{1}{\rho} dp + d\sigma \rightarrow \text{(ID)}$$

(ID) holds subject to the condition that t is a constant.

Integrating and rearranging,

$$\int d\left(\frac{\partial \phi}{\partial t}\right) - \frac{1}{2} \int d(\nabla \phi)^2 = \int \frac{1}{\rho} dp + \int d\sigma$$

$$\Rightarrow \frac{\partial \phi}{\partial t} - \frac{1}{2} (\nabla \phi)^2 = \int \frac{dp}{\rho} + \sigma$$

$$\frac{1}{2} (\vec{q}^2) + \sigma + \int \frac{dp}{\rho} - \frac{\partial \phi}{\partial t} = f(t) \rightarrow \text{(IE)}$$

where $\vec{q} = -\nabla \phi$ and $f(t)$ is an arbitrary constant arising out of

integration, a fn of t where t is a constant.

Eqn (IE) is called Bernoulli's eqn in the most general form.

When the flow is steady,

$$\frac{\partial \phi}{\partial t} = 0 \text{ & } f(t) = \text{constant} = c \text{ (say)}$$

then (IE) becomes,

$$\frac{1}{2} (\vec{q})^2 + \sigma + \int \frac{dp}{\rho} = c \rightarrow (IF)$$

Further if the fluid is homogeneous and incompressible then σ is a constant and (IF) becomes,

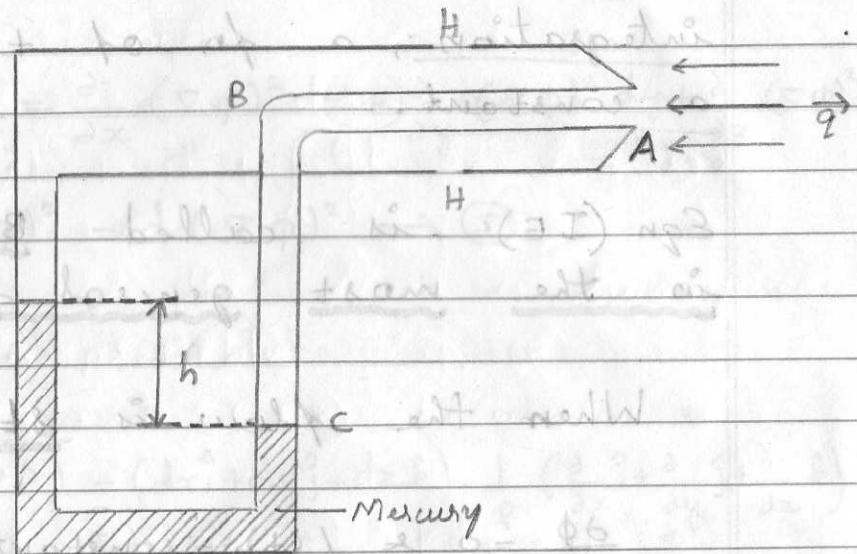
$$\frac{1}{2} (\vec{q})^2 + \sigma + \int \frac{dp}{\rho} = c$$

$$\text{i.e.) } \frac{1}{2} (\vec{q})^2 + \sigma + \frac{p}{\rho} = c \rightarrow (IG)$$

Applications : (eg ①, pg. 100)

1. Pitot Tube:

Pitot tube is a simple device used to measure the velocity of the fluid flow at any point in the fluid regions.



Suppose it is desired to measure the velocity \vec{q} of a stream of water.

It is a tube of very small cross section bent in the form of right angle and kept vertical.

To measure velocity \vec{q} of stream of water flowing, the inner tube BA in the apparatus is placed upstream of the flow. The outer tube contains holes such as H.

Let p denote the pressure of the stream where the fluid velocity is \vec{q} then p is also considered the pressure on inside and outside of the hole and hence ^{also} at the ^{mercury} point D of the mercury level in the U-tube.

Consider a stream entering the inner tube AB, this is brought to rest at the mercury level at c.

Let p_0 be the pressure at c. Flow being steady applying Bernoulli's eqn along the stream line extending to infinity, entering the inner tube at A and finally being brought to rest at c.

General form of Bernoulli's eqn is given by,

$$\frac{1}{2}(\vec{q}^2) + \rho g + \int \frac{dp}{\rho} - \frac{\partial \phi}{\partial t} = f(t) \rightarrow (*)$$

where $\vec{q} = -\nabla \phi$ & $f(t)$ is an arbitrary constant, a fn of t where t is a constant.

Since the flow is of potential type,

$$\vec{q} = -\nabla \phi$$

and since body forces are conservative

$$\vec{F} = -\nabla \phi$$

When the flow is steady,

$$\frac{\partial \phi}{\partial t} = 0, f(t) = \text{constant } c \text{ (say)}$$

Then (*) becomes,

$$\frac{1}{2} (\vec{q})^2 + \sigma + \int \frac{dp}{\rho} = c \rightarrow (***)$$

Further if the fluid is homogeneous and incompressible then ρ is a constant and (**) becomes,

$$\frac{1}{2} (\vec{q})^2 + \sigma + \frac{1}{\rho} \int dp = 0$$

At c,

$$\text{i.e. } \frac{1}{2} (\vec{q})^2 + \sigma + \frac{p_0}{\rho} = c \rightarrow ①$$

Applying Bernoulli's eqn to the water level at D,

the flow comes to rest at D due to the effect of the mercury level.

In the limiting case, let \vec{q} denote the velocity and p the pressure at D, then we have

$$\frac{1}{2} \vec{q}^2 + \frac{p}{\rho} + \sigma = \text{constant at D} \rightarrow ②$$

At c,

$$|\vec{q}| = |\vec{o}| = 0 \text{ and } p = p_0$$

(1) & (2) now

(1) becomes,

$$\frac{P_0}{\rho} + \sigma = \text{constant at } c \rightarrow (1A)$$

geneousFrom (2) & (1A), (2) - (1A) \rightarrow

$$\frac{1}{2} \vec{g}^2 + \frac{P}{\rho} + \cancel{\sigma} - \frac{P_0}{\rho} - \cancel{\sigma} = \text{const at D} -$$

const on c

$$\frac{1}{2} \vec{g}^2 = \frac{P_0 - P}{\rho} + c$$

$$\vec{g}^2 = \frac{2(P_0 - P)}{\rho} + c$$

the

$$|\vec{g}| = \sqrt{\frac{2(P_0 - P)}{\rho}} + c$$

③

at D
veryThe pressure difference $P_0 - P$ is
measured from the difference in levels
of mercury.

If h is the height difference
b/w the 2 level and if σ is the
density of the mercury then σgh
is the pressure difference, σ is pressure

D \rightarrow ②

$$\sigma gh = P_0 - P \rightarrow (4)$$

where g is the gravitation force.

From ③ & ④,

$$|\vec{q}| = + \sqrt{\frac{2\sigma gh}{\rho}} + c$$

which gives the speed of stream of water.

Thus pitot tube can be used to compute the speed of fluid flow which finds its applications in the following areas,

Suitable to measure sub-sonic air airstreams.

At low speeds air can be treated as an incompressible fluid for similar purposes of analysis.

Steady Motion under conservative body forces:

Consider

The flow is an inviscid flow, body forces are conservative, there exist scalar potential $\nabla \cdot \mathbf{r}$ such that $\mathbf{F} = -\nabla \cdot \mathbf{r} \rightarrow (*)$ and the flow is irrotational, hence there exist velocity potential $\nabla \phi$ such that $\vec{q} = -\nabla \phi \rightarrow (**)$

9 to Euler's eqn:

mass flow rate is \dot{m} + exit to atmosphere

$$\text{and flow } \frac{d\vec{q}}{dt} = \vec{F} - \frac{\nabla P}{\rho} \quad [\text{from (I)}]$$

[from (IA)]

$$\Rightarrow \frac{\partial \vec{q}}{\partial t} + \nabla \left(\frac{1}{2} \vec{q}^2 \right) - \vec{q} \times (\nabla \times \vec{q}) = - \frac{\nabla P}{\rho} + \vec{F}$$

$$\boxed{\vec{F} = -\nabla \phi}, \text{ because } (*)$$

Flow is steady,

$$\frac{\partial \vec{q}}{\partial t} = \vec{0} \quad \& \quad \nabla \times \vec{q} = \epsilon \vec{s}$$

$$\Rightarrow \nabla \left(\frac{1}{2} (\vec{q})^2 \right) - \vec{q} \times (\vec{s}) = - \frac{\nabla P}{\rho} + \nabla \phi$$

(Also,

$$d\left(\frac{P}{\rho}\right) = \frac{1}{\rho} dP + \rho d\left(\frac{1}{\rho}\right)$$

$$\Rightarrow \frac{dP}{\rho} = d\left(\frac{P}{\rho}\right)$$

$$\nabla \int \frac{dP}{\rho} = \nabla \int d\left(\frac{P}{\rho}\right) = \nabla \left(\frac{P}{\rho}\right)$$

$$\Rightarrow \nabla \left(\frac{1}{2} \vec{q}^2 + \int \frac{dP}{\rho} + \phi \right) = \vec{q} \times \vec{s} \rightarrow (1)$$

so water won't move

① If \vec{r} is the position vector of P at time t during the small increase of time in δt seconds, $\delta \vec{r}$ will be the infinitesimal displacement in the fluid element.)

(Scalar multiplying eqn ① by $d\vec{x}$) which would be the limiting case representing a time independent variation of the p.v \vec{x}

② (i.e.) as $\delta t \rightarrow 0$, $\frac{\delta \vec{r}}{\delta t} = d\vec{r}$)

We have,

$$① \cdot d\vec{x} \Rightarrow \nabla \left(\frac{1}{2} \vec{q}^2 + \int \frac{dp}{\rho} + \phi \right) \cdot d\vec{x} = (\vec{q} \times \vec{L}) d\vec{x}$$

$$\begin{aligned} ① & \quad \left[\because \nabla \phi \cdot d\vec{r} = \left(\left(\frac{\partial \phi}{\partial x} \hat{i} + \frac{\partial \phi}{\partial y} \hat{j} + \frac{\partial \phi}{\partial z} \hat{k} \right) \phi \right) \right] \\ & \quad (dx\hat{i} + dy\hat{j} + dz\hat{k}) \\ & \quad = \frac{\partial \phi}{\partial x} dx + \frac{\partial \phi}{\partial y} dy + \frac{\partial \phi}{\partial z} dz \\ & \quad = d\phi \end{aligned}$$

$$\therefore d \left(\frac{1}{2} \vec{q}^2 + \int \frac{dp}{\rho} + \phi \right) = d\vec{r} \cdot (\vec{q} \times \vec{L})$$

↳ ②

Hence we have various case

of P

increase
will be
so the

Case (i) :

(ii) ~~zero~~

$$\nabla \times \vec{q} = \vec{0} \text{ where } \vec{q} \neq \vec{0} \quad (\text{i.e. } \nabla \times \vec{q} \neq \vec{0})$$

then \vec{q} is parallel to $\vec{\xi}$ i.e., stream lines are parallel to vortex lines. and ~~coincide~~

\Rightarrow stream lines and vortex lines are coincides for such a motion \vec{q} .
is called Beltrami vector.

(2) becomes,

$$d\left(\frac{1}{2}\vec{q}^2 + \int \frac{dp}{\rho} + \alpha\right) = d\vec{r} \cdot (\vec{q} \times \vec{\xi})$$

$$= d\vec{r} \cdot \vec{0}$$

$$d\left(\frac{1}{2}\vec{q}^2 + \int \frac{dp}{\rho} + \alpha\right) = \vec{0} \rightarrow (3)$$

By integrating (3), we get

$$\boxed{\frac{1}{2}\vec{q}^2 + \int \frac{dp}{\rho} + \alpha = \text{constant}} \rightarrow (3A)$$

(3A) holds throughout the entire field of flow. The constant is the same as the differential $d\vec{r}$ is an arbitrarily a small variation of position vector \vec{r} in the field.

Case (ii)

($\vec{v} + \vec{\omega} \times \vec{r}$, vorticity) If $\vec{q} \times \vec{s} = 0$ where $\vec{s} = \vec{o}$ (i.e., $\nabla \times \vec{q} = \vec{0}$)

i.e.) the flow is irrotational.

Eqn (2) becomes

$$\begin{aligned} d\left(\frac{1}{2}\vec{q}^2 + \int \frac{db}{e} + \alpha\right) &= d\vec{s} \cdot (\vec{q} \times \vec{s}) \\ &= d\vec{s} \cdot \vec{o} \\ &= \vec{o} \rightarrow (4) \end{aligned}$$

By integrating (4), we get

$$\boxed{\frac{1}{2}\vec{q}^2 + \int \frac{db}{e} + \alpha = \text{constant}} \rightarrow (4A)$$

(4A) holds throughout the entire field of flows, the constant is the same as the differential $d\vec{s}$ is an arbitrarily small variation of position vector \vec{s} in the field.

Case (iii):

If the fluid is homogeneous and the flow is irrotational

i.e.) fluid is incompressible.

④ A becomes ② primitive

$$\boxed{\frac{1}{2} \vec{q}^2 + \frac{p}{\rho} + \sigma = \text{constant}} \rightarrow ④ B$$

Case (iv): one ② pd. besides wh

Suppose $\vec{q} \times \vec{s} \neq \vec{0}$

$$\text{Consider } \vec{q} \cdot (\vec{q} \times \vec{s}) = [\vec{q} \quad \vec{q} \quad \vec{s}] = 0$$

$$\Rightarrow \vec{q} \perp (\vec{q} \times \vec{s})$$

$$\text{Also } \vec{s} \cdot (\vec{q} \times \vec{s}) = [\vec{s} \quad \vec{q} \quad \vec{s}] = 0$$

$$\Rightarrow \vec{s} \perp (\vec{q} \times \vec{s})$$

If \vec{s} is so chosen such that
~~($\vec{q} \times \vec{s}$)~~ $\cdot d\vec{s} = \vec{0}$, this happens whenever
 $d\vec{s}$ lies in the plane of (\vec{q}) and (\vec{s}) .

Thus we take the variation $d\vec{s}$ in
 the surface containing both stream lines
 and vortex lines, we have

$$\therefore d \left(\frac{1}{2} \vec{q}^2 + \int \frac{dp}{\rho} + \sigma \right) = 0 \rightarrow ⑤$$

Integrating (5), we have (5B)

$$\frac{1}{2} \vec{q}^2 + \int \frac{dp}{\rho} + \sigma = \text{constant} \rightarrow (5B)$$

Further note that over surface described by (5), we have (5B), the surface containing stream lines and vortex lines. The constant appearing in (5B) remains everywhere on the surface which varies from one to another on sets of surfaces.

(5B) is irrespective of the fact whether the motion is irrotational or not.

If the flow is homogeneous and the fluid under consideration is incompressible, then we have

$$(5B) \Rightarrow \frac{1}{2} \vec{q}^2 + \frac{1}{\rho} \int dp + \sigma = \text{constant}$$

$$\Rightarrow \frac{1}{2} \vec{q}^2 + \frac{p}{\rho} + \sigma = \text{constant} \rightarrow (6)$$

To establish Kelvin's theorem expressing the condition under which circulation in a closed circuit of fluid particles moving along with the fluid is constant.

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8m

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20m

Some Further Aspects of Vortex Motion



Kelvin's theorem:

(5B)

(X)

Statement:

An

(A) For an inviscid fluid circulation around any closed circuit of fluid particles, moving along with the fluid at time t , remains constant provided the body forces are conservative and the pressure is a single valued function of density only.

Proof:

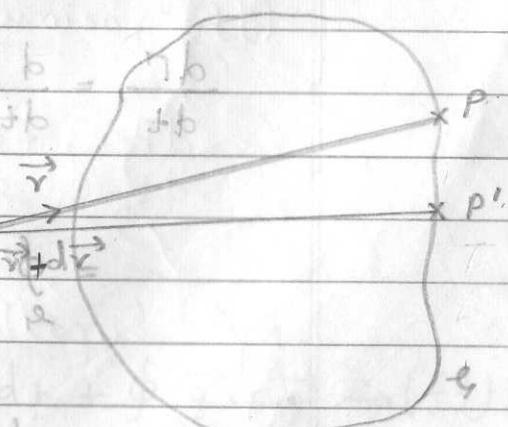
[Write 1st part of Euler's eqn of motion]

Consider a closed circuit Γ . Since the body forces are

Since the body forces are conservative,

$\Rightarrow \exists$ a scalar pt

$$\Rightarrow \vec{F} = -\nabla \varphi$$



Since the flow is of potential kind,

$\Rightarrow \exists$ a vector potential ϕ

$$\Rightarrow \vec{q} = -\nabla \phi$$

∴ The Euler's eqn of motion is given by

$$\frac{d\vec{q}}{dt} = \vec{F} - \frac{1}{\rho} \nabla p \rightarrow (I)$$

$$\Rightarrow \frac{d\vec{q}}{dt} = -\nabla u - \frac{1}{\rho} \nabla p \rightarrow (IA)$$

Let P' be a neighbouring point to P on Γ with position vector $\vec{x} + d\vec{x}$

Circulation around the closed circuit (Γ) is given by

$$\boxed{\Gamma = \oint_{\Gamma} \vec{q} \cdot d\vec{x}}$$

The time derivative of Γ following the fluid flow is

$$\frac{d\Gamma}{dt} = \frac{d}{dt} \oint_{\Gamma} \vec{q} \cdot d\vec{x}$$

$$= \oint_{\Gamma} \frac{d}{dt} (\vec{q} \cdot d\vec{x}) \quad [As it is finite \phi \text{ and time derivative are interchangeable}]$$

$$= \oint_{\Gamma} \left(\frac{d}{dt} \vec{q} \cdot d\vec{x} + \vec{q} \cdot \frac{d}{dt} (d\vec{x}) \right)$$

$$= \oint_{\Gamma} \frac{d}{dt} \vec{q} \cdot d\vec{x} + \oint_{\Gamma} \vec{q} \cdot \frac{d}{dt} (d\vec{x})$$

$$= \oint_{\Gamma} \left(-\nabla \cdot \mathbf{a} - \frac{1}{\rho} \nabla p \right) \cdot d\vec{\mathbf{x}} \quad (\text{from IA})$$

$$+ \oint_{\Gamma} \vec{q} \cdot \frac{d}{dt} (d\vec{\mathbf{x}})$$

$$\text{i.e.) } \frac{d\Gamma}{dt} = - \oint_{\Gamma} \underbrace{\nabla \cdot \mathbf{a} \cdot d\vec{\mathbf{x}}}_{(1)} - \oint_{\Gamma} \underbrace{\frac{1}{\rho} \nabla p \cdot d\vec{\mathbf{x}}}_{(2)} + \oint_{\Gamma} \vec{q} \cdot \frac{d}{dt} (d\vec{\mathbf{x}})$$

$\hookrightarrow (f)$

$$(1) \Rightarrow \nabla \cdot \mathbf{a} \cdot d\vec{\mathbf{x}} = \left(\frac{\partial}{\partial x} \hat{i} + \frac{\partial}{\partial y} \hat{j} + \frac{\partial}{\partial z} \hat{k} \right) \cdot \mathbf{a} \cdot (dx\hat{i} + dy\hat{j} + dz\hat{k})$$

$$= \frac{\partial}{\partial x} a_x dx + \frac{\partial}{\partial y} a_y dy + \frac{\partial}{\partial z} a_z dz$$

$$= d \cdot \mathbf{a}$$

$$(2) \Rightarrow \frac{1}{\rho} (\nabla p \cdot d\vec{\mathbf{x}}) = \frac{1}{\rho} \left(\frac{\partial}{\partial x} \hat{i} + \frac{\partial}{\partial y} \hat{j} + \frac{\partial}{\partial z} \hat{k} \right) p \cdot (dx\hat{i} + dy\hat{j} + dz\hat{k})$$

$$= \frac{1}{\rho} \left(\frac{\partial}{\partial x} (p) dx + \frac{\partial}{\partial y} (p) dy + \frac{\partial}{\partial z} (p) dz \right)$$

$$= \frac{1}{\rho} \frac{dp}{dt}$$

$\therefore (f)$ becomes

$$\frac{d\Gamma}{dt} = - \oint_{\Gamma} \mathbf{a} \cdot d\vec{\mathbf{x}} - \oint_{\Gamma} \frac{1}{\rho} dp + \oint_{\Gamma} \vec{q} \cdot \frac{d}{dt} (d\vec{\mathbf{x}}) \rightarrow (ff)$$

(3)

We can show that,

$$\frac{d}{dt} (d\vec{\mathbf{x}}) = d \left(\frac{d\vec{\mathbf{x}}}{dt} \right) \rightarrow (*)$$

$$(ff) \Rightarrow \frac{d\Gamma}{dt} = -\phi d_{-2} - \phi \frac{1}{e} dp + \frac{1}{2} \phi d(\vec{q})^2$$

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(3) >

$$\text{Also, } \vec{q} \cdot d\vec{q} = \frac{1}{2} d(\vec{q})^2$$

$$[\text{Also } \vec{q} \cdot \vec{q} = \vec{q}^2]$$

$$d(\vec{q} \cdot \vec{q}) = d\vec{q} \cdot \vec{q} + \vec{q} \cdot d\vec{q}$$

$$= d(\vec{q}^2)$$

$$\rightarrow d(\vec{q} \cdot \vec{q}) = \frac{1}{2} d(\vec{q}^2)$$

We have,

$$-\phi d_{-2} = -2 \quad \text{defined}$$

$$\text{and } \phi d(\vec{q})^2 = (\vec{q})^2 \quad \text{defined}$$

(ff) becomes to,

$$\frac{d\Gamma}{dt} = -\phi \frac{1}{e} dp \rightarrow (fff)$$

By hypothesis,

p is a single valued function dependent on ϕ only.

$$\text{Hence } -\phi \frac{1}{e} dp = -\phi d\left(\frac{p}{e}\right) = 0$$

III - fin U

~~so that~~ i.e. $(\nabla \cdot \vec{f})$ reduces to ~~out and~~

~~exterior to the~~ $\Rightarrow \frac{d \Gamma}{dt} = 0$ ~~wall and~~ (i.e.)

wall $\rightarrow \Gamma$ constant w.r.t. moving circuit

Now to prove $(*)$ is also ~~valid~~

i.e.) $\frac{d}{dt} (\vec{d} \vec{x}) = \oint_C d \left(\frac{d \vec{r}}{dt} \right)$

At time t , the position vectors of points P & P' which reference to the origin O respectively are \vec{r} and $\vec{r} + d\vec{r}$.

Then at time $t+dt$, their position vectors are

$$\vec{r} + \frac{d\vec{r}}{dt} dt, (\vec{r} + d\vec{r}) + \frac{d}{dt} (\vec{r} + d\vec{r}) dt \rightarrow ①$$

Since $d\vec{x}$ denotes the variation of fluid element at p to that at p' , the position vector of p' at time $t+dt$ is given by

$$(\vec{r} + d\vec{r}) + d(\vec{r} + \frac{d\vec{r}}{dt} dt) \rightarrow ②$$

Comparing ① & ② carefully we notice that $(*)$ holds.