

Hydrodynamics and elasticity, Lecture notes

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1 | Basic definitions

1.1 Organization

Course web page: <https://www.fuw.edu.pl/~mklis/hydro2022.html>

Requirements to obtain credit:

- Homework (30%),
- Midterm exam (35%),
- Written exam (35%),
- Oral exam (optional, only improves).

1.2 Basic laws

Example 1.2.1. Out of context Navier-Stokes equations:

$$\frac{\partial \underline{u}}{\partial t} + \underline{u} \cdot \nabla \underline{u} = -\frac{1}{\rho} \nabla p + \nu \nabla^2 \underline{u},$$

$$\nabla \cdot \underline{u} = 0.$$

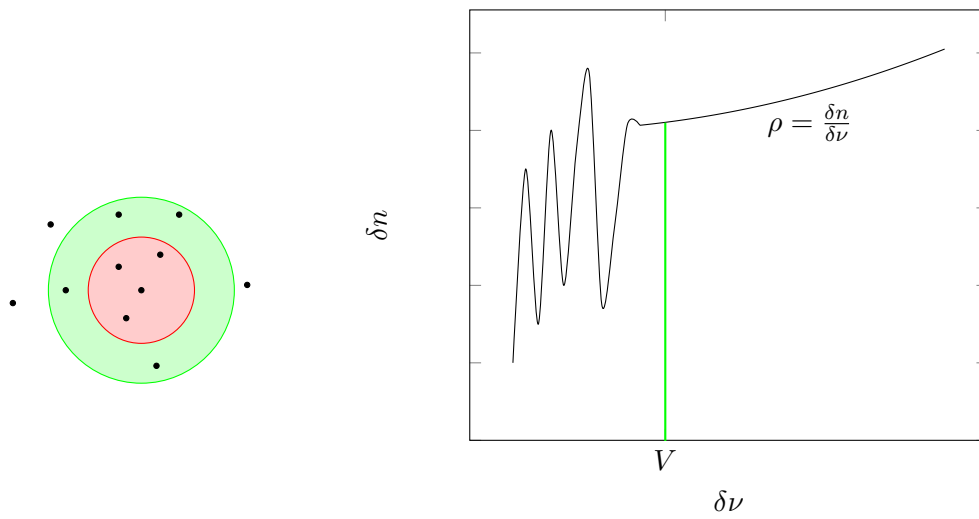
where $\underline{u}(\underline{r}, t)$ is a fluid velocity vector field, $\rho(\underline{r}, t)$ is a fluid density, $p(\underline{r}, t)$ is a pressure, ν is a kinematic viscosity.

Continuum hypothesis states that

$$\rho = \frac{\delta \eta}{\delta \nu},$$

where η is a number of particles in a region and ν is a volume of this region. Of course if the volume ν is small enough ρ may vary a lot (obviously it may not even be continuous). There is however such volume V which is „big enough”, so that for $\nu > V$ ρ does not vary „that much”.

Since matter is not continuous, we can't speak of a density at a point (in a mathematical sense) and thus, when we use the phrase „point” we mean „at a point for homogeneous physical system”.

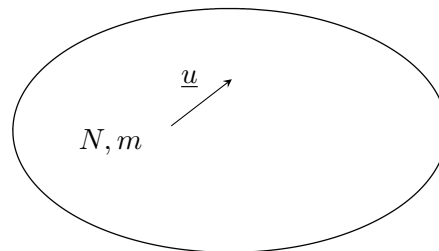
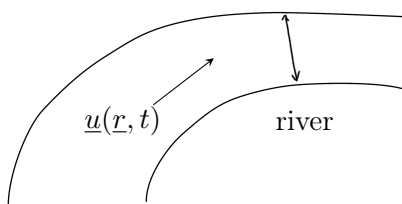


We also introduce length scale separation

$$\text{molecular length scale} \ll \nu^{\frac{1}{3}} \ll L.$$

We call those characteristic length scales „*micro*”, „*muso*” and „*macro*” respectively.

1.2.1 Equilibrium thermodynamics



Let us examine an example of a water in a river. Assume that the water has some velocity \underline{u} , momentum \underline{P} and also that it can exchange mass and heat, the latter given by $dQ = TdS$. Therefore an incremental change in the energy of this system can be expressed as

$$dE = \underline{u} \cdot d\underline{P} - p dV + T dS + \mu dN,$$

where μ is the chemical potential of the system. Thus

$$E = E(\underline{P}, V, S, N).$$

This is the **energy representation** of a thermodynamical system. Comparing the equations above we obtain

$$dE = \underbrace{\frac{\partial E}{\partial \underline{P}}}_{\underline{u}} d\underline{P} + \underbrace{\frac{\partial E}{\partial V}}_{-p} dV + \underbrace{\frac{\partial E}{\partial S}}_T dS + \underbrace{\frac{\partial E}{\partial N}}_{\mu} dN,$$

Those are called the Gibbs relation for E .

If we want to compute it for a fixed entropy we get

$$dS = \frac{1}{T} dE - \frac{\underline{u}}{T} d\underline{P} + \frac{p}{T} dV - \frac{\mu}{T} dN.$$

Thus

$$S = S(E, \underline{P}, V, N), \quad dS = \underbrace{\frac{\partial S}{\partial E}}_{\frac{1}{T}} dE + \underbrace{\frac{\partial S}{\partial \underline{P}}}_{-\frac{\underline{u}}{T}} d\underline{P} + \underbrace{\frac{\partial S}{\partial V}}_{\frac{p}{T}} dV + \underbrace{\frac{\partial S}{\partial N}}_{-\frac{\mu}{T}} dN,$$

Those are Gibbs relations for S .

It is very tricky to control entropy — it's much easier to control the temperature. To obtain a description of our system when T is an independent variable we need to use another thermodynamical potential, which is Helmholtz free energy. Transition is obtained by

$$(S \rightarrow T) \quad F = E - TS, \quad F = F(\underline{P}, V, T, N)$$

$$dF = -S dT - p dV + \underline{u} d\underline{P} + \mu dN.$$

Now we can do the same to switch other variables. Thus we obtain

$$(V \rightarrow P) \quad H = E + pV, \quad H = H(\underline{P}, p, S, N),$$

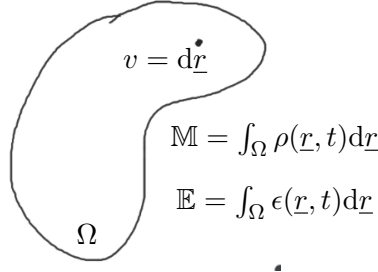
which is called an enthalpy,

$$(S \rightarrow T) \quad G = E + pV - TS, \quad G = G(\underline{P}, p, T, N),$$

which is called a Gibbs potential.

Now we want to consider if they are Galilean invariant

$$E(\underline{P}, V, S, N) = E_0(V, S, N) + \frac{\underline{P}^2}{2M},$$



Rysunek 1.3: Sample volume Ω . ρ stands for mass density, ϵ for density of the system.

where E_0 is the **internal energy**. For other potentials we obtain

$$H(\underline{P}, p, S, N) = H_0(p, S, N) + \frac{P^2}{2M},$$

$$F(\underline{P}, V, T, N) = F_0(T, V, N) + \frac{P^2}{2M},$$

$$G(\underline{P}, V, T, N) = G_0(T, p, N) + \frac{P^2}{2M}.$$

Using $\underline{P} = M\underline{u}$ we get

$$E = E_0 + \frac{1}{2}M\underline{u}^2.$$

$$dE = dE_0 + \underline{u} \cdot d\underline{P},$$

$$dS = -\frac{1}{T}\underline{u} \cdot d\underline{P} + \frac{1}{T}dE + \frac{p}{T}dV - \frac{\mu}{T}dN,$$

$$dS = \frac{1}{T}dE_0 + \frac{p}{T}dV - \frac{\mu}{T}dN \implies S(\underline{P}, E, V, N) = S(E_0, V, N).$$

Thus S is a Galilean invariant.

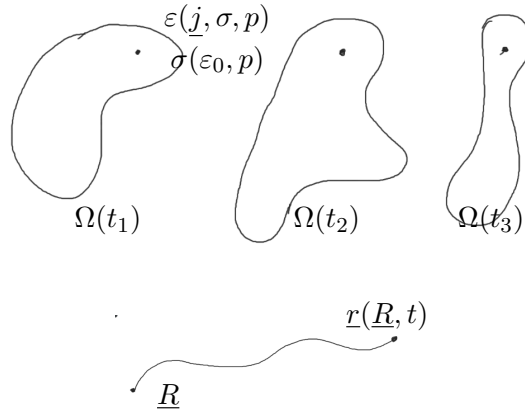
1.2.2 Heterogenous macroscopic system

„Densities” are extensive properties per unit volume. Assume that $V = \text{const}$, $dV = 0$.
Example of densities

- $\rho = \frac{M}{V}$ mass density,
- $\underline{j} = \frac{\underline{P}}{V}$ momentum density,
- $\epsilon = \frac{E}{V}$ energy density,
- $\sigma = \frac{S}{V}$ entropy density.

$$dE = \underline{u}d\underline{P} - pdV + TdS + \mu dN / \cdot \frac{1}{V}.$$

$$d\epsilon = \underline{u} \cdot d\underline{j} + Td\sigma + \mu dn, \quad dn = \frac{d\rho}{m},$$



$$\rho = \frac{M}{V} = \frac{Nm}{V} = nm \implies dn = \frac{d\rho}{m}.$$

Thus the energy fundamental representation in terms of densities can be written as

$$\varepsilon = \varepsilon(\underline{j}, \sigma, \rho).$$

After performing a Galilean transform we get

$$\varepsilon(\underline{j}, \sigma, \rho) = \varepsilon_0(\sigma, \rho) + \frac{1}{2}\rho \underline{u}^2.$$

We can do the same to represent entropy in terms of densities

$$dS = \dots \frac{1}{V} \implies d\sigma = \frac{1}{T}d\varepsilon_0 - \frac{1}{T}\frac{\mu}{m}dp.$$

Thus

$$\sigma = \sigma(\varepsilon_0, \rho),$$

which is also Galilean invariant.

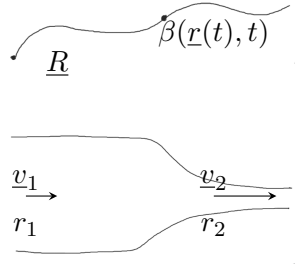
1.2.3 Flow of Heterogenous macroscopic system

We are using thermodynamics equilibrium despite the fact that the system flows (which means that it is *not* in the equilibrium). However there is no contradiction if we assume local, thermodynamical equilibrium. The same assumption is made in Navier-Stokes equations.

From now on we use pseudostatic transition. The difference between quasi-static and pseudostatic is that pseudostatic need not to be reversible. Forces which acts during quasi-static transformation are those which keeps the system in equilibrium. Example of pseudostatic transition which is not quasi-static is a flow of viscous fluid.

1.2.4 Kinematics

We have two descriptions: Eulerian and Lagrangian. Transition between them is obtained by



the solutions to the initial problem

$$\frac{d\mathbf{r}}{dt} = \mathbf{u}(\mathbf{r}, t), \quad \mathbf{r}(t=0) = \mathbf{R}.$$

We want to find dependence of densities as the particle move, and those are

	Euler	Lagrange
velocity	$\mathbf{u}(\mathbf{r}, t)$	$\mathbf{u}[\mathbf{r}(t), t]$
density	$\rho(\mathbf{r}, t)$	$\rho[\mathbf{r}(t), t]$

$$\frac{d\mathbf{u}}{dt} = ?, \quad \frac{d\rho}{dt} = ?.$$

To do that define $\beta = (\mathbf{u}, \rho, S, \sigma, \dots)$. We want to find how β change while following the motion of the particle $\mathbf{r}(t)$.

$$\begin{aligned} \frac{d\beta}{dt} &= \frac{d\beta[\mathbf{r}(t), t]}{dt} = \frac{\partial \beta}{\partial t} + \frac{dr_1}{dt} \frac{\partial \beta}{\partial r_1} + \frac{dr_2}{dt} \frac{\partial \beta}{\partial r_2} + \frac{dr_3}{dt} \frac{\partial \beta}{\partial r_3} \\ &= \frac{\partial \beta}{\partial t} + \mathbf{u}(t) \cdot \nabla \beta = \frac{\partial \beta}{\partial t} + d\beta(\mathbf{u}(t)). \end{aligned}$$

Equation

$$\frac{d\beta}{dt} = \frac{\partial \beta}{\partial t} + \mathbf{u}(t) \cdot \nabla \beta, \quad (1.1)$$

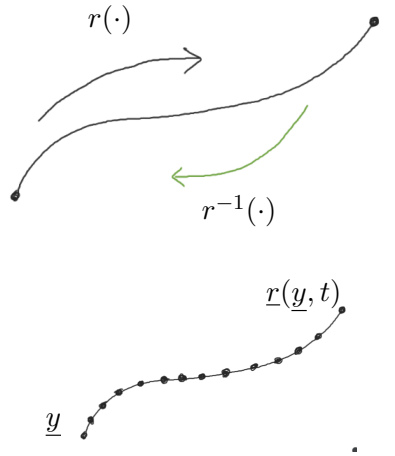
shows a relationship between Eulerian and Lagrangian world, because $\frac{d\beta}{dt}$ is a typical Lagrangian while fields (like $\mathbf{u}(t)$) are common in Euler's description. (Fields are Eulerian objects). We introduce a **total (material) derivative** as

$$\frac{d\dots}{dt} = \underbrace{\frac{\partial \dots}{\partial t}}_{\text{local derivative}} + \underbrace{\mathbf{u} \cdot \nabla(\dots)}_{\text{advective derivative}}.$$

Acceleration of fluid particle

$$\beta = \mathbf{u} \implies \frac{d\beta}{dt} = \frac{d\mathbf{u}}{dt}.$$

Consider converging channel with a stationary flow, i.e. $\frac{\partial \mathbf{u}}{\partial t} = 0$.



Rysunek 1.4: Streakline is a line made of all particles that for time s , $0 \leq s \leq t$ passed through a fixed point \underline{y} .

With that in mind

$$\frac{d\underline{u}}{dt} = \frac{\partial \underline{u}}{\partial t} + \underline{u} \cdot \nabla \underline{u} = \underline{u} \cdot \nabla \underline{u}.$$

The term $\underline{u} \cdot \nabla \underline{u}$ should be interpreted as follows. Treat \underline{u} as a map $\underline{u}(\underline{r}) : \mathbb{R}^3 \rightarrow \mathbb{R}^3$. Thus $\nabla \underline{u}$ is just a map $D\underline{u} : \mathbb{R}^3 \rightarrow \mathbb{R}^3$ expressed by a matrix

$$\underline{u}(\underline{r}) = \begin{bmatrix} u_1(r_1, r_2, r_3) \\ u_2(r_1, r_2, r_3) \\ u_3(r_1, r_2, r_3) \end{bmatrix}, \quad D\underline{u} = \begin{bmatrix} \frac{\partial u_1}{\partial r_1} & \frac{\partial u_1}{\partial r_2} & \frac{\partial u_1}{\partial r_3} \\ \frac{\partial u_2}{\partial r_1} & \frac{\partial u_2}{\partial r_2} & \frac{\partial u_2}{\partial r_3} \\ \frac{\partial u_3}{\partial r_1} & \frac{\partial u_3}{\partial r_2} & \frac{\partial u_3}{\partial r_3} \end{bmatrix}.$$

Therefore inner product $\underline{u} \cdot \nabla \underline{u}$ really means

$$\underline{u} \cdot \nabla \underline{u} = (D\underline{u})(\underline{u}).$$

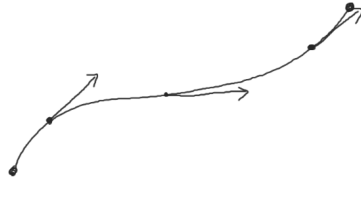
At least I think so...

The term $\underline{u} \cdot \nabla \underline{u}$ is called a **convective acceleration**. Although the flow is stationary, the particle experiences acceleration related to movement along the flow lines.

We can interpret the fluid flow as a mapping which takes one point and maps it to the other.

Streakline Imagine a cigarette and assume that there is no diffusion. The smoke is made out of small, fluid particles. The flow line is made out of fluid particles which were passing through \underline{y} in the time interval $0 \leq s \leq t$. Suppose that we froze time at $t = t_0$. Choose point A . **Streakline** through the point A is a curve made of all particles (in the given moment) that have passed through the point A at some $t < t_0$.

$$\underline{r}[\underline{R}(\underline{y}, s), t], \quad 0 \leq s \leq t.$$



Rysunek 1.5: Streamline is just a flow of a vector field for at a fixed time t .

Streamline Streamline is a integral curve of a vector field $X(t)$ at a given time $t = t_0$. They do not intersect neither with each other nor with themselves. Equation:

$$\frac{dr}{ds} = \underline{u}(\underline{r}, t).$$

Trajectory Trajectory is a path traced by a chosen particle.

For the stationary flow the streakline and streamline are the same.

For stationary flows i.e. $(\frac{\partial \underline{u}}{\partial t} = 0)$. Trajectory \equiv streakline \equiv streamline.

1.3 Balance equations

$\underline{u}(\underline{r}, t)$ — velocity field, $\rho(\underline{r}, t)$ — density flow. They are not completely independent since the mass has to be conserved.

$$\begin{aligned} M &= \int_V \rho(\underline{r}, t) d\underline{r}, \\ \frac{\partial M}{\partial t} &= \frac{\partial}{\partial t} \int_V \rho(\underline{r}, t) d\underline{r} = \int_V \frac{\partial \rho(\underline{r}, t)}{\partial t} d\underline{r}, \\ \frac{\partial M}{\partial t} &= \int_V \frac{\partial \rho}{\partial t} d\underline{r} = - \int_{\partial V} \rho \underline{u} \cdot \underline{n} da, \end{aligned}$$

using Stokes theorem

$$= - \int_V \nabla \cdot (\rho \underline{u}) d\underline{r}.$$

In other words

$$\int_V \left[\frac{\partial \rho}{\partial t} + \nabla \cdot (\rho \underline{u}) \right] d\underline{r} = 0,$$

and, since V is arbitrary,

$$\frac{\partial \rho}{\partial t} + \nabla \cdot (\rho \underline{u}) = 0.$$

Which is called the **continuity equation**. This is the general mass conservation, fluid can change density and so on.

2 | Lecture 2

Reminder Material derivative

$$\frac{d}{dt} = \frac{D}{Dt} = \frac{\partial}{\partial t} + \underline{u} \cdot \nabla.$$

Mass conservation implies the continuity equation

$$\frac{\partial \rho}{\partial t} + \nabla \cdot (\rho \underline{u}) = 0.$$

New stuff Expanding the above equation we obtain

$$\underbrace{\frac{\partial \rho}{\partial t} + \underline{u} \cdot \nabla \rho}_{\frac{d\rho}{dt}} + \rho \nabla \cdot \underline{u} = 0,$$

and thus

$$\frac{1}{\rho} \frac{d\rho}{dt} = -\nabla \cdot \underline{u}.$$

If we introduce **specific volume** $\nu = 1/\rho$ we obtain

$$\frac{1}{\nu} \frac{d\nu}{dt} = \nabla \cdot \underline{u}.$$

We introduced that because we want to study incompressible flow. If the flow is incompressible we express it by saying that

$$\frac{d\rho}{dt} = 0 \quad \text{or} \quad \frac{d\nu}{dt} = 0.$$

From the continuity equation incompressibility of the flow implies that

$$\nabla \cdot \underline{u} = 0.$$

For the incompressible flow the \underline{u} is divergence-free or solenoidal (**TODO**I didn't hear well).

If $\nabla \cdot \underline{u} = 0$ and $f(\underline{r}, t) = f_0 = \text{const}$ then¹

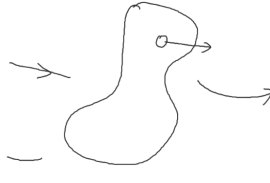
$$\forall t \quad f(\underline{r}, t) = f_0 = \text{const}.$$

Positive divergence implies expansion, negative implies compression.

¹It follows from the continuity equation.

2.1 Newton's second law (Momentum balance)

Consider a closed system (volume $V(t)$) which is comprised of the same fluid particles (it flows with a fluid).



We want to calculate the momentum of such **material volume**. It is obviously an integral

$$\underline{P}(t) = \int_{V(t)} \rho(\underline{r}, t) \underline{u}(\underline{r}, t) d\underline{r}.$$

That is a linear momentum of the material volume. We want to state the Newton second law:

$$\frac{d\underline{P}(t)}{dt} = \underline{F},$$

where \underline{F} is the net force.

$$\frac{d\underline{P}}{dt} = \frac{d}{dt} \int_{V(t)} \rho(\underline{r}, t) \underline{u}(\underline{r}, t) d\underline{r} = ?.$$

Here is a theorem (i.e. fancy name for Leibniz rule):

Theorem 2.1.1 (Raynold's transport theorem).

$$\frac{d}{dt} \int_{V(t)} \beta(\underline{r}, t) d\underline{r} = \int_V \left[\frac{\partial \beta}{\partial t} + \nabla \cdot (\beta \underline{u}) \right] d\underline{r} = \int_V \left[\frac{\partial \beta}{\partial t} + \underline{u} \cdot \nabla \beta + \beta \nabla \cdot \underline{u} \right],$$

where V is a fixed quantity, called control volume (i.e. any volume that coincides with $V(t)$).

The things that contribute to this change can be interpreted as

1. local change $\frac{\partial \beta}{\partial t}$,
2. advection i.e. $\underline{u} \cdot \nabla \beta$,
3. changing volume i.e. $\beta \nabla \cdot \underline{u}$.

Applying RTT to the momentum we get

$$\frac{d\underline{P}}{dt} = \frac{d}{dt} \int_{V(t)} \rho \underline{u} d\underline{r} = \int_V \left[\frac{\partial \rho \underline{u}}{\partial t} + \nabla \cdot (\rho \underbrace{\underline{u} \underline{u}}_{\underline{u} \otimes \underline{u}}) \right]$$

Homework Show that if $\beta = \rho b$, then

$$\frac{d}{dt} \int_{V(t)} \rho b d\underline{r} = \int_V \rho \frac{db}{dt} d\underline{r},$$

using RTT (Raynold's transport theorem) and the continuity equation.

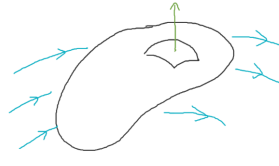
$$\frac{\xi}{x} = \int_V dV$$

$$\frac{d\underline{P}}{dt} = \frac{d}{dt} \int_{V(t)} \rho \underline{u} d\underline{r} = \int_V \rho \frac{d\underline{u}}{dt} d\underline{r} = \underline{F},$$

and thus the integral's form of Newton second law

$$\int_V \rho \frac{d\underline{u}}{dt} d\underline{r} = \underline{F}.$$

2.2 Further consequences of RTT



$$M = \int_V \rho d\underline{r} \implies \frac{\partial M}{\partial t} = \frac{\partial}{\partial t} \int_V \rho d\underline{r} = \int_V \frac{\partial \rho}{\partial t} d\underline{r} = - \int_V \nabla \cdot (\rho \underline{u}) d\underline{r} = - \int_{\partial V} \rho \underline{u} \cdot \hat{n} dS.$$

To note: material volume is the volume that flows with the fluid.

Consider a material volume $V(t)$ and its mass given by

$$M = \int_{V(t)} \rho d\underline{r}.$$

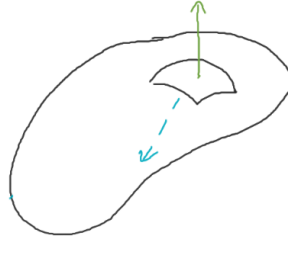
The mass conservation means that

$$\frac{dM}{dt} = 0.$$

We calculate

$$\frac{dM}{dt} = \frac{d}{dt} \int_{V(t)} \rho d\underline{r} = \int_V \left[\frac{\partial \rho}{\partial t} + \nabla \cdot (\rho \underline{u}) \right] d\underline{r} = 0.$$

The above equation states that the mass of the material volume which travels with the flow remains constant.



Force model In the following equation

$$\int_V \rho \frac{d\mathbf{u}}{dt} d\mathbf{r} = \mathbf{F},$$

we do not know what \mathbf{F} is and therefore need a model for it.

Consider that the fluid acts on its surface element da (with a normal vector \hat{n}). Let \mathbf{t} be a force per unit area, and $d\mathbf{F} = \mathbf{t}da$.

$$\mathbf{F} \stackrel{\text{model}}{=} \int_{\partial V} d\mathbf{F} = \int_{\partial V} \mathbf{t}da = - \int_V \nabla p d\mathbf{r}.$$

Assume that $\mathbf{t} = -p\hat{n}$, where p is a pressure and ∇p a **pressure field**.

$$\begin{aligned} \int_V \rho \frac{d\mathbf{u}}{dt} d\mathbf{r} &= - \int_V \nabla p d\mathbf{r}, \\ \int_V \left[\rho \frac{d\mathbf{u}}{dt} + \nabla p \right] d\mathbf{r} &= 0 \implies \rho \frac{d\mathbf{u}}{dt} = -\nabla p. \end{aligned}$$

We may also write it as

$$\rho \left(\frac{\partial \mathbf{u}}{\partial t} + \mathbf{u} \cdot \nabla \cdot \mathbf{u} \right) = -\nabla p.$$

This is the **Euler model of the ideal fluid** (ideal fluid without dissipation).

2.3 Equilibrium

Equilibrium is obtained when $\mathbf{u} = 0$ and thus $\nabla p = 0$ (especially if $p = \text{const}$).

- ideal fluid $\mathbf{t} = -p\hat{n}$
- In general $\mathbf{t} = \underline{\underline{\Sigma}}^T \cdot \hat{n}$, where $\underline{\underline{\Sigma}}$ is a **Cauchy stress tensor** (second order tensor).

In general case force can have a form

$$\mathbf{F} = \int_{\partial V} \mathbf{t}da = \int_{\partial V} \underline{\underline{\Sigma}}^T \cdot \hat{n}da = \int_V \nabla \cdot \underline{\underline{\Sigma}} d\mathbf{r},$$

with the stress tensor

$$\underline{\underline{\Sigma}} = -p \cdot \underline{\underline{1}} + \underline{\underline{\Sigma}}'.$$

Newton's second law

$$\int_V \rho \frac{d\mathbf{u}}{dt} d\mathbf{r} = \int \nabla \cdot \underline{\underline{\Sigma}} d\mathbf{r}.$$

$$\underline{\underline{\Sigma}} = \underbrace{-p\mathbf{1}}_{\text{ideal term}} + \underbrace{\underline{\underline{\Sigma}}}_{\text{deviatoric part}}$$

Deviatoric part vanishes in equilibrium.

Ideal fluid model $\underline{\underline{\Sigma}}' = 0$, $\underline{\underline{\Sigma}} = -p\mathbf{1}$.

$$\nabla \cdot \underline{\underline{\Sigma}} = \nabla \cdot (-p\mathbf{1}) = -\nabla p.$$

Summary Til now we formulated

- Continuity equation

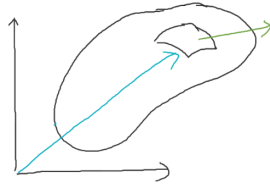
$$\frac{\partial \rho}{\partial t} + \nabla \cdot (\rho \mathbf{u}) = 0$$

- Newton's law

$$\rho \left(\frac{\partial \mathbf{u}}{\partial t} + \mathbf{u} \cdot \nabla \cdot \mathbf{u} \right) = \nabla \cdot \underline{\underline{\Sigma}}$$

- What's next? Angular momentum.

2.4 Angular momentum



Consider a material volume $V(t)$, with density ρ and a point at \mathbf{r} moving with a velocity \mathbf{u} .

$$\underline{\underline{L}}(t) = \int_{V(t)} \mathbf{r} \times \rho \mathbf{u} d\mathbf{r} = \int_{V(t)} \rho (\mathbf{r} \times \mathbf{u}) d\mathbf{r} = \int_{V(t)} \rho \mathbf{l} d\mathbf{r},$$

where $\mathbf{l} = \mathbf{r} \times \mathbf{u}$ is the **angular momentum per unit mass**.

The law of the change of the angular momentum

$$\frac{d\underline{\underline{L}}}{dt} = \underline{\underline{N}},$$

where $\underline{\underline{N}}$ is a net torque acting on $V(t)$.

$$\frac{d\underline{\underline{L}}}{dt} = \frac{d}{dt} \int_{V(t)} \rho \mathbf{l} d\mathbf{r} \stackrel{\text{RTT} + \text{cont}}{=} \int_V \rho \frac{d\mathbf{l}}{dt} d\mathbf{r}.$$

We calculate \underline{N} by

$$\underline{N} = \int_{\partial V} \underline{r} \times \underline{t} da.$$

$$\int_V \rho \frac{d\underline{l}}{dt} d\underline{r} = \int_{\partial V} \underline{r} \times \underline{t} da = \int_{\partial V} (\underline{r} \times \Sigma^T) \cdot \hat{n} da$$

using divergence theorem

$$= \int_V \nabla \cdot [(\underline{r} \times \Sigma^T)^T] d\underline{r}. \quad (2.1)$$

where the divergence theorem reads

$$\int_{\partial V} T \cdot \hat{n} da = \int_V \nabla \cdot T^T d\underline{r}.$$

Going back to 2.1

$$= \int_V [\underline{r} \times \nabla \cdot \Sigma - 2\underline{\sigma}] d\underline{r},$$

where $\underline{\sigma}$ is the axial vector associated with Σ . Thus

$$\int_V \rho \frac{d\underline{l}}{dt} d\underline{r} = \int_V [\underline{r} \times \nabla \cdot \Sigma - 2\underline{\sigma}] d\underline{r},$$

and since the volume V can be anything we get

$$\rho \frac{d\underline{l}}{dt} = \underline{r} \times \nabla \cdot \Sigma - 2\underline{\sigma}. \quad (2.2)$$

This is too complicated, we need to simplify it.

$$\rho \frac{d\underline{l}}{dt} = \rho \frac{d\underline{r} \times \underline{u}}{dt} = \rho \underline{r} \times \frac{d\underline{u}}{dt} + \rho \frac{d\underline{r}}{dt} \times \underline{u} = \underline{r} \times \underbrace{\rho \frac{d\underline{u}}{dt}}_{\nabla \cdot \underline{\Sigma}} = \underline{r} \times \nabla \cdot \underline{\Sigma}.$$

Substituting it to 2.2 we get

$$\underline{\sigma} = 0.$$

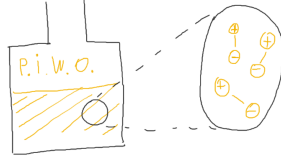
Thus the Σ is symmetric.

The stress tensor need not to be symmetric for magnetic fluids (it works for simple fluids).

„Complex” fluids Consider a magnetic fluid in a bottle, with magnetic dipoles. Assume that we apply a magnetic field, so there is a reorientation and an **internal torque** appears.

$$\underline{N} = \int_{\partial V} \underline{r} \times \underline{t} da + \int_V \underline{b} d\underline{r},$$

where \underline{b} is the internal torque.



2.5 Energy conservation

Consider a material volume V and a small surface element da . The energy is given by

$$E(t) = \int_{V(t)} \rho(\underline{r}, t) e(\underline{r}, t) d\underline{r},$$

where $e(\underline{r}, t)$ is the energy for unit mass.

The „law” of change

$$\frac{dE}{dt} = \frac{dW}{dt} + \frac{dQ}{dt},$$

where W is a mechanical work and Q is a heat.

Using RTT and continuity we get

$$\frac{dE}{dt} = \frac{d}{dt} \int_{V(t)} \rho e d\underline{r} = \int_V \rho \frac{de}{dt} d\underline{r},$$

$$\frac{dW}{dt} = \int_{\partial V} \underline{t} \cdot \underline{u} da = \int_{\partial V} (\underline{\underline{\Sigma}}^T \cdot \underline{u}) \cdot \hat{n} da = \int_V \nabla \cdot (\underline{\underline{\Sigma}} \cdot \underline{u}) d\underline{r}.$$

$$\frac{dQ}{dt} = - \int_{\partial V} \underline{q} \cdot \hat{n} da = - \int_V \nabla \cdot \underline{q} d\underline{r},$$

where \underline{q} is the heat flow per unit surface per unit time.

Summing up we get

$$\rho \frac{de}{dt} = \nabla \cdot (\underline{\Sigma} \cdot \underline{u}) - \nabla \cdot \underline{q}. \quad (2.3)$$

Let's introduce the separation

$$e = e_0 + \frac{1}{2} \underline{u}^2.$$

Substituting it into 2.3 we obtain

$$\rho \frac{de_0}{dt} = -\rho \frac{d}{dt} \left(\frac{1}{2} \underline{u}^2 \right) + \nabla \cdot (\underline{\Sigma} \cdot \underline{u}) - \nabla \cdot \underline{q}.$$

$$\rho \frac{d}{dt} \left(\frac{1}{2} \underline{u}^2 \right) = \underline{u} \cdot \rho \frac{d\underline{u}}{dt} = \underline{u} \cdot (\nabla \cdot \underline{\Sigma}) = \nabla \cdot (\underline{\Sigma} \cdot \underline{u}) - \underline{\Sigma} : \nabla \underline{u}$$

For the internal energy e_0 :

$$\rho \frac{de_0}{dt} = \Sigma : \nabla \underline{u} - \nabla \cdot \underline{q}.$$

For $\Sigma = -p\underline{1} + \Sigma'$

$$\Sigma : \nabla \underline{u} = -p(\nabla \cdot \underline{u}) + \Sigma' : \nabla \underline{u}.$$

2.6 Ideal fluid approximation

We assume no deviatoric stress, and no heat flows

$$\Sigma' = 0$$

$$\underline{q} = 0.$$

For the ideal fluid

$$\rho \frac{de_0}{dt} = -p(\nabla \cdot \underline{u}).$$

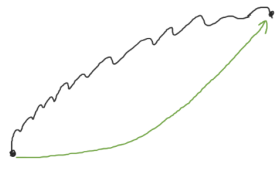
This approximation is a result of assuming that the particles move all together, so there is no viscosity and also no way to conduct a heat.

For a compressible flow

- $\nabla \cdot \underline{u} > 0 \implies \frac{de_0}{dt} < 0$ (expansion),
- $\nabla \cdot \underline{u} < 0 \implies \frac{de_0}{dt} > 0$ (compression),

and for the incompressible flow there is no way to change internal energy, $\frac{de_0}{dt} = 0$. In ideal fluid there is no dissipation. However the real fluids do.

2.7 Entropy balance



Consider a closed system and assume that a heat has been transferred to the system. We can move between S and $S + dS$ by reversible and irreversible paths. For the reversible one we have

$$dS = \frac{dQ}{T},$$

and for an irreversible proces

$$dS \geq \frac{dQ}{T}.$$

$$S(t) = \int_{V(t)} \rho(\underline{r}, t) s(\underline{r}, t) d\underline{r},$$

where $s(\underline{r}, t)$ is entropy per unit volume. Entropy balance implies

$$\begin{aligned}\frac{dS}{dt} &= \frac{d_e S}{dt} + \frac{d_i S}{dt} \\ d_e S &= \frac{dQ}{T}. \\ \frac{dS}{dt} &= \frac{d}{dt} \int_{V(t)} \rho s d\underline{r} = \int -\rho \frac{ds}{dt} d\underline{r}, \\ \frac{d_e S}{dt} &= - \int_{\partial V} \frac{q}{T} \cdot \hat{n} da = - \int_V \nabla \cdot \left(\frac{q}{T} \right) d\underline{r} \\ \frac{d_i S}{dt} &= \int_V \theta d\underline{r},\end{aligned}$$

where θ is the entropy production per unit volume per unit time. Thus the **entropy balance equation** is

$$\rho \frac{ds}{dt} = -\nabla \cdot \left(\frac{q}{T} \right) + \theta, \quad \theta \geq 0.$$

For ideal fluid (no internal effects) $\theta = 0$, and the change of entropy

$$\frac{ds}{dt} = 0.$$

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Recall

$$\begin{aligned}\frac{\partial \rho}{\partial t} + \nabla \cdot (\rho \underline{u}) &= 0, \\ \frac{1}{\nu} \frac{d\nu}{dt} &= \nabla \cdot \underline{u}, \quad \nu = \frac{1}{\rho}, \\ \rho \frac{d\underline{u}}{dt} &= \rho \left(\frac{\partial \underline{u}}{\partial t} + \underline{u} \cdot \nabla \underline{u} \right) = \nabla \cdot \underline{\Sigma} + \rho \underline{g},\end{aligned}\tag{3.1}$$

angular momentum balance

$$\underline{\Sigma}^T = \underline{\Sigma}.$$

energy per unit mass

$$\rho \frac{de_0}{dt} = -p(\nabla \cdot \underline{u}) + \underline{\Sigma}' : \nabla \underline{u} - \nabla \cdot \underline{q}\tag{3.2}$$

entropy per unit mass

$$\rho \frac{ds}{dt} = -\nabla \cdot \left(\frac{\underline{q}}{T} \right) + \theta,$$

where θ is the **entropy production**. The second law of thermodynamics says that $\theta > 0$.

Those are complete system of balance equations and in fact quite general ones.

Local thermodynamics equilibrium approximation

$$\theta = \rho \frac{ds}{dt} + \nabla \cdot \left(\frac{\underline{q}}{T} \right),$$

$$s = s(e_0, \nu),$$

and the Gibbs relation

$$ds = \frac{1}{T} de_0 + \frac{p}{T} d\nu.$$

de_0/dt comes from the flow of a energy and the $d\nu/dt$ part comes from continuity equation. We can write it as

$$\rho \frac{ds}{dt} = \frac{1}{T} \underbrace{\rho \frac{de_0}{dt}}_{\text{Eq 3.2}} + \frac{p}{T} \underbrace{\frac{1}{\nu} \frac{d\nu}{dt}}_{\text{Eq 3.1}},$$

$$\rho \frac{ds}{dt} = \frac{1}{T} [-p(\nabla \cdot \underline{u}) + \underline{\Sigma}' : \nabla \underline{u} - \nabla \cdot \underline{q}] + \frac{p}{T} (\nabla \cdot \underline{u}).$$

$$\nabla \cdot \left(\frac{\underline{q}}{T} \right) = \frac{1}{T} \nabla \cdot \underline{q} - \frac{1}{T^2} \underline{q} \cdot \nabla T,$$

$$\theta = -\cancel{\frac{p}{T}(\nabla \cdot \underline{u})} + \frac{1}{T} \Sigma' : \nabla \underline{u} - \cancel{\frac{1}{T} \nabla \cdot \underline{q}} + \cancel{\frac{p}{T}(\nabla \cdot \underline{u})} + \cancel{\frac{1}{T} \nabla \cdot \underline{q}} - \frac{1}{T^2} \underline{q} \cdot \nabla T.$$

Thus the entropy production is given by the formula

$$\theta = -\frac{1}{T^2} \underbrace{\underline{q} \cdot \nabla T}_{\text{heat flux}} + \frac{1}{T} \underbrace{\Sigma' : \nabla \underline{u}}_{\text{momentum flux}} \geq 0.$$

Imagine that you have a flow and a temperature gradient — then the fluid will flow from the hotter part to the colder. Note that both terms have to be positive, because when one is missing, the other one must be non-negative.

How does those equations simplify in the ideal fluid model?

3.1 Ideal fluid model

Recall that for the ideal fluid we have

$$\underline{q} = 0, \quad \Sigma' = 0 \quad \implies \quad \theta = 0,$$

and therefore there is no entropy production.

Hydrodynamics of ideal fluids:

1. $\frac{\partial \rho}{\partial t} + \nabla \cdot (\rho \underline{u}) = 0,$
2. $\rho \frac{d\underline{u}}{dt} = -\nabla p + \rho \underline{g},$
3. $\rho \frac{de_0}{dt} = -p(\nabla \cdot \underline{u}),$
4. $\frac{ds}{dt} = 0.$

Since the 2nd equation is a vector one we have 6 equations. Unfortunately, there are 7 unknowns, which means we are one equation short. The one that is missing is the thermodynamical equation of state (for pressure).

3.2 Thermodynamics

In the equation

$$e_0 = e_0(s, \nu)$$

the specific volume as a variable is not that interesting, since it cannot be easily controlled. It would be much better to use a pressure instead of ν . To do that we switch the thermodynamical potential, from energy to enthalpy

$$h_0 = e_0 + p\nu = e_0 + \frac{p}{\rho}, \quad h_0 = h_0(s, p),$$

$$dh_0 = Tds + \nu dp = \underbrace{Tds}_{=0} + \frac{dp}{\rho},$$

if we work in a regime where s is fixed. Thus

$$dh_0 = \frac{dp}{\rho}, \quad h_0 = h_0(p), \quad \frac{\partial h_0}{\partial p}|_s = \frac{1}{\rho}.$$

Therefore $\rho = \rho(p)$ or $p = p(\rho)$ and that's what we've been missing. This equation is called the **equation of state**.

For an ideal gas

$$p(\rho) = C\rho^\gamma, \quad \gamma = \frac{c_p}{c_v}.$$

For an ideal fluid

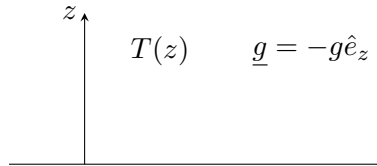
$$\rho = \rho_0 = \text{const.}$$

3.3 Hydrostatics

Now that we have a complete set of equations, we can solve physical problems. Assume for now that $\underline{u} = 0$ which leads to

$$\nabla p = \rho \underline{g}, \quad p = p(\rho).$$

Assume that we work with an ideal gas and know the temperature profile i.e. $T(\text{height})$.



$$\frac{dp}{dz} = -\rho g, \quad p(z) = ?,$$

For an ideal gas $p = \rho RT$, thus

$$\int_{p(z=)}^{p(z)} \frac{dp'}{p'} = - \int_0^z \frac{g dz}{RT(z)} \implies p(z) = p(0) \exp \left(- \int_0^z \frac{g dz}{RT(z)} \right),$$

where we've used

1. $T(z) \rightarrow p(z)$
2. $\rho(z)$ from the equation $\rho = \frac{p}{RT}$

Sound waves Fluid in equilibrium (neglecting gravity)

$$\underline{u} = 0,$$

and $p = p_{og} = \text{const.}$, $\rho = \rho_{og} = \text{const.}$, $s = s_{og} = \text{const.}$. A sound wave is just a „small” perturbation of the equilibrium state

$$\underline{u}(\underline{r}, t) = 0 + \underline{u}'(\underline{r}, t) \quad (|\underline{u}'| \ll c)$$

$$\begin{aligned}
p(\underline{r}, t) &= p_{eq} + p'(\underline{r}, t) \quad (p' \ll p_{eq}) \\
\rho(\underline{r}, t) &= \rho_{eq} + \rho'(\underline{r}, t) \quad (\rho' \ll \rho_{eq}), \\
s(\underline{r}, t) &= s_{eq} = \text{const.}
\end{aligned}$$

Now we want to find primed variables. How? By using the equations for an ideal fluid. We have

$$\frac{\partial \rho}{\partial t} + \nabla \cdot (\rho \underline{u}) = 0, \quad (3.3)$$

$$\rho \left(\frac{\partial \underline{u}}{\partial t} + \underline{u} \cdot \nabla \underline{u} \right) = -\nabla p, \quad (3.4)$$

$$s = s_{eq} = \text{const.} \quad (3.5)$$

The sound velocity squared

$$\nabla p = \underbrace{\left(\frac{dp}{d\rho} \right)}_{c^2} \nabla \rho = c^2 \nabla \rho.$$

Those equations are very hard to solve mathematically, since they are non-linear. However, since we just want to solve easier case, those can be linearly approximated. Omitting all second order terms, we obtain

$$\frac{\partial(\rho_{eq} + p')}{\partial t} + \nabla \cdot [(\rho_{eq} + p') \underline{u}] = 0,$$

and the **linearized continuity equation**

$$\nabla \cdot \underline{u}' = -\frac{1}{\rho_{eq}} \frac{\partial \rho'}{\partial t}. \quad (3.6)$$

Now we take care of the second equation (3.4)

$$(\rho_{eq} + \rho') \left[\frac{\partial \underline{u}'}{\partial t} + \underbrace{\underline{u}' \cdot \nabla \underline{u}'}_{\approx 0} \right] = -c^2 \nabla(\rho_{eq} + \rho'),$$

and obtain the **linearized equation of motion**

$$\frac{\partial \underline{u}'}{\partial t} = -\frac{c^2}{\rho_{eq}} \nabla \rho'.$$

We have the following system

$$\begin{aligned}
\nabla \cdot \underline{u}' &= -\frac{1}{\rho_{eq}} \frac{\partial \rho'}{\partial t}, \\
\frac{\partial \underline{u}'}{\partial t} &= -\frac{c^2}{\rho_{eq}} \nabla \rho'.
\end{aligned}$$

Differentiating the first equation with respect to time and using divergence operator on the second one we get

$$\frac{\partial^2(\rho')}{\partial t^2} = c^2 \nabla^2 \rho'.$$

So the speed of sound

$$c^2 = \left(\frac{\partial p}{\partial \rho} \right)_s.$$

For $s = \text{const}$, $p(\rho) = C\rho^\gamma$, it can be approximated as

$$\begin{aligned} \left(\frac{\partial p}{\partial \rho} \right)_s &= \gamma \frac{p}{\rho} \approx \gamma \frac{p_{eq}}{\rho_{eq}}. \\ c^2 &\approx \gamma \frac{p_{eq}}{\rho_{eq}} = \gamma R T_{eq}. \end{aligned}$$

1-D wave equation

$$\frac{\partial^2(\rho')}{\partial t^2} = c^2 \frac{\partial^2(\rho')}{\partial x^2}, \quad \rho'(x, t) = ?. \quad (3.7)$$

To solve it we write ρ' in a Fourier representation

$$\rho'(x, t) = \int dk \int d\omega \rho'(k, \omega) e^{ikx} e^{-i\omega t}.$$

Substituting the above into 3.7 we get

$$\begin{aligned} (-i\omega)^2 \rho'(k, \omega) &= c^2 (ik)^2 \rho'(k, \omega), \\ \omega^2 &= c^2 k^2 \implies \omega = \pm ck. \end{aligned}$$

The above equation is called the **dispersion relation**. Using it we get

$$\exp[i(kx - \omega t)] = \exp[ik(x - ct)].$$

Finally

$$\rho'(x, t) = \int dk \rho'_1(x) \exp(ik(x - ct)) + \int dk \rho'_2 \exp(ik(x + ct)),$$

which are two families of perturbations travelling in the opposite directions.

3D case Using Fourier representation

$$\begin{aligned} \rho'(\underline{r}, t) &= \int d\underline{k} \int d\omega \rho'(\underline{k}, \omega) \exp[i(\underline{k} \cdot \underline{r} - \omega t)], \\ \frac{\partial^2(\rho')}{\partial t^2} &= c^2 \nabla^2 \rho', \\ (-i\omega)^2 \rho'(\underline{k}, \omega) - c^2 (i\underline{k})^2 \rho'(\underline{k}, \omega) &\implies \omega = \pm c |\underline{k}|, \\ \exp[i(\underline{k} \cdot \underline{r} - \omega t)] &= \exp[i\underline{k}(\hat{\underline{k}} \cdot \underline{r} - ct)]. \end{aligned}$$

Note that c doesn't depend on ω . If it would, we will call such wave **dispersive**. Nondispersive waves travel without changing shape.

We know $\rho'(\underline{r}, t)$. What about $p'(\underline{r}, t)$ and $\underline{u}'(\underline{r}, t)$?

By substituting into

$$dp = \left(\frac{p}{\rho} \right)_s d\rho,$$

relations $p'(\underline{r}, t) = c^2 \rho'(\underline{r}, t)$, we can calculate $p'(\underline{r}, t)$.

To obtain velocity \underline{u} we note that

$$\frac{\partial \underline{u}'}{\partial t} = -\frac{c^2}{\rho_{eq}} \nabla \rho'$$

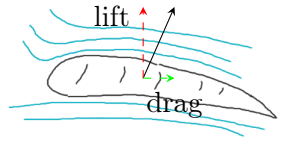
and, by using Fourier transform, we obtain

$$\begin{aligned} \underline{u}'(\underline{k}, \omega) &= \frac{c^2}{\rho_{eq} \omega} \underline{k} \rho'(\underline{k}, \omega) = \pm \frac{c^2}{\rho_{eq} c k} \underline{k} \rho'(\underline{k}, \omega) \\ \underline{u}' &= \pm \frac{c}{\rho_{eq}} \rho' \hat{k}. \end{aligned}$$

Sound waves are longitudinal (particles move in the same way the waves propagates).

3.4 Aerodynamics

Imagine that there is a wing profile and consider a wind tunnel configuration.



Question: what is the force that the flow exerts on the object? The force can be decomposed into two parts: the lift and the drag.

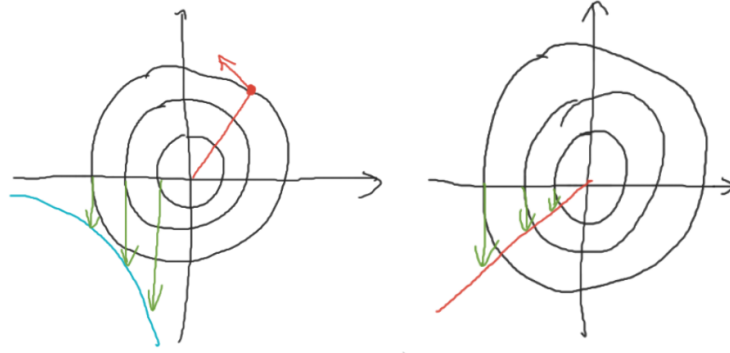
$$\underline{F} = - \int_S p \hat{n} ds, \quad p(\underline{r}, t) = ?$$

To obtain p , \underline{u} and \underline{F} we have to solve Euler's equations

$$\begin{aligned} \rho \left(\frac{\partial \underline{u}}{\partial t} + \underline{u} \cdot \nabla \underline{u} \right) &= -\nabla p + \rho \underline{g} \\ \frac{\partial p}{\partial t} + \rho \cdot (\rho \underline{u}) &= 0. \end{aligned}$$

Those are most often almost impossible to solve. We should try to simplify the mathematical problem here.

First we can try choosing a different variable. We change \underline{u} into **vorticity** $\underline{\xi} = \nabla \times \underline{u}$.



Euler's equations

$$\frac{\partial \underline{u}}{\partial t} + \underline{u} \cdot \nabla \cdot \underline{u} = -\frac{1}{\rho} \nabla p + \underline{g}$$

1. $\underline{g} = -\nabla \chi$

2.

$$\underline{u} \cdot \nabla \cdot \underline{u} = \nabla \left(\frac{u^2}{2} \right) + (\nabla \times \underline{u}) \times \underline{u} = \nabla \left(\frac{u^2}{2} \right) + \underline{\xi} \times \underline{u}$$

3. $\frac{1}{\rho} \nabla p = \nabla h_0$

$$\frac{\partial \underline{u}}{\partial t} + \nabla \times \underline{u} = -\nabla h_0 - \nabla \left(\frac{u^2}{2} \right) - \nabla \chi = -\nabla \left(h_0 + \frac{1}{2} u^2 \chi \right) =: -\nabla \mathfrak{h}, \quad (3.8)$$

where

$$\mathfrak{h} = h_0 + \frac{u^2}{2} + \chi.$$

Then Equation 3.8 reads

$$\frac{\partial \underline{u}}{\partial t} + \underline{\xi} \times \underline{u} = -\nabla \mathfrak{h}.$$

By taking the rotation of both sides

$$\overbrace{\frac{\partial \nabla \times \underline{u}}{\partial t}}^{\underline{\xi}} + \nabla \times (\underline{\xi} \times \underline{u}) = 0,$$

we obtain

$$\nabla \times (\underline{\xi} \times \underline{u}) = (\underline{u} \cdot \nabla) \underline{\xi} - (\underline{\xi} \cdot \nabla) \underline{u} + \xi(\nabla \cdot \underline{u}) - \underline{u}(\nabla \cdot \underline{\xi}) =$$

if the flow is incompressible

$$\frac{\partial \underline{\xi}}{\partial t} + \underline{u} \cdot \nabla \underline{\xi} = \underline{\xi} \cdot \nabla \cdot \underline{u}$$

$$\frac{d \underline{\xi}}{dt} = \underline{\xi} \cdot \nabla \underline{\xi}.$$

If $\nabla \cdot \underline{u} = 0$ the $\underline{\xi}$ is an invariant of a motion.