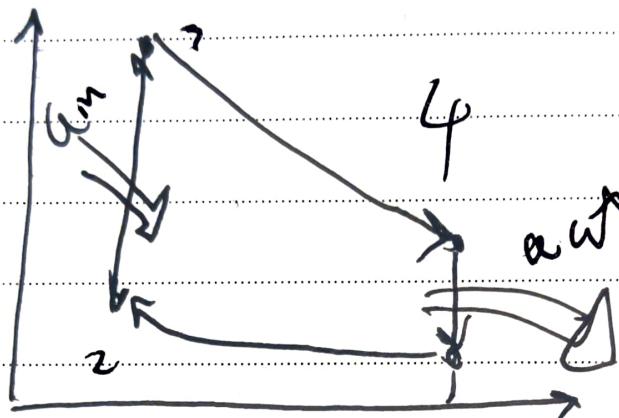


Problem set 6

a) Otto Cycle

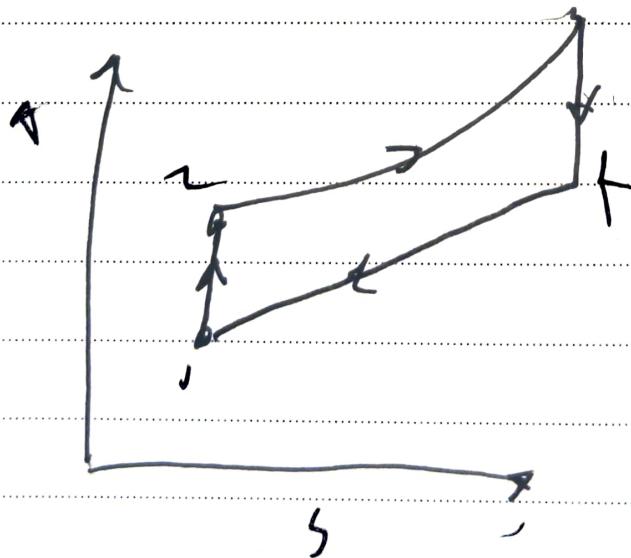


1-2 Isentropic compression

2-3 Isochoric heating

3-4 Isentropic expansion

4-1 Isochoric heat rejection

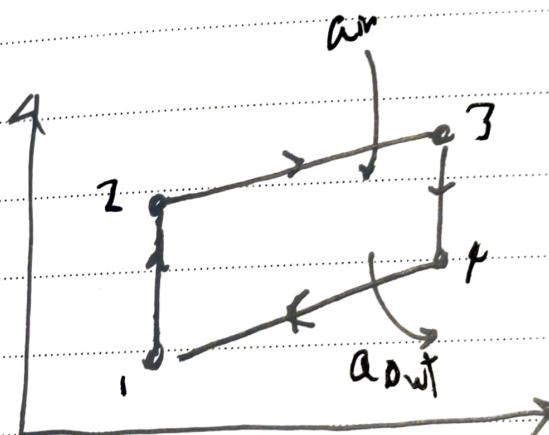
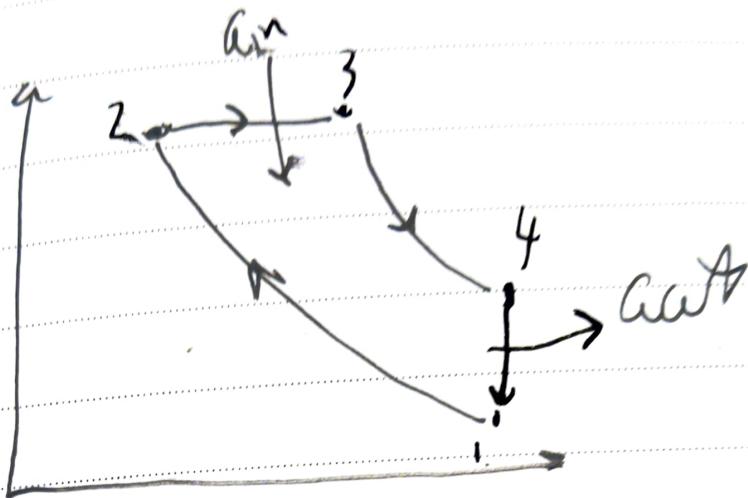


Assumptions: Cross-sections - fixed, Perfect gases

- DCE & DPE are negligible

3rd law of thermodynamics is applicable

b) Diesel cycle



Assumptions: Standard air & steady
DKE & DPE are negligible

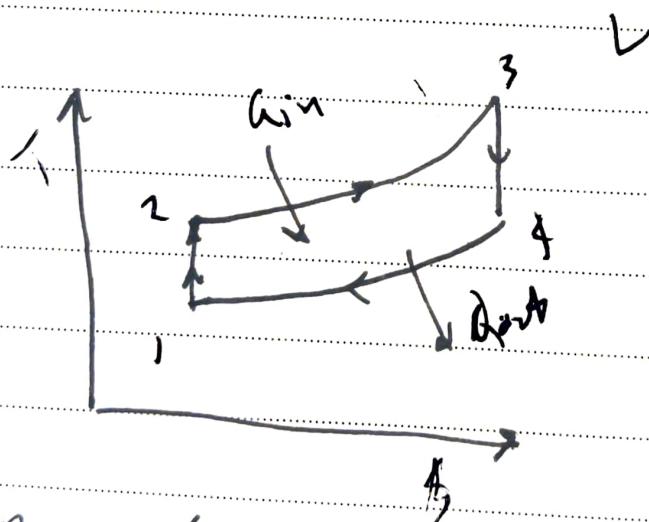
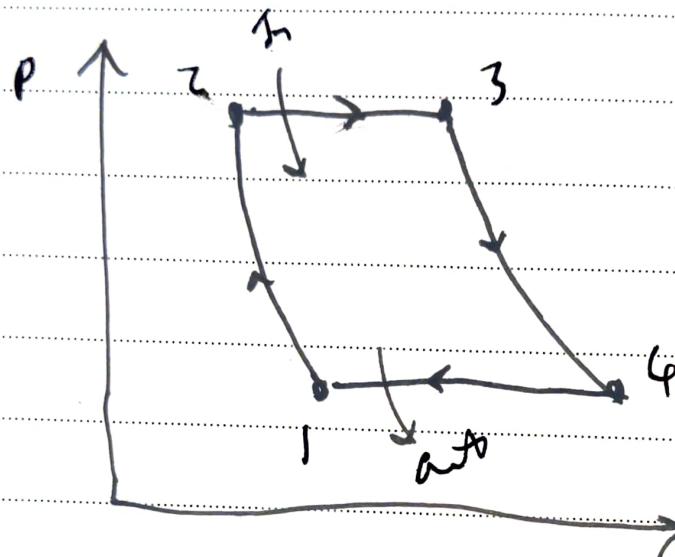
1-2 Isentropic compression

2-3 ~~constant~~ Isentropic compression

3-4 Isentropic expansion

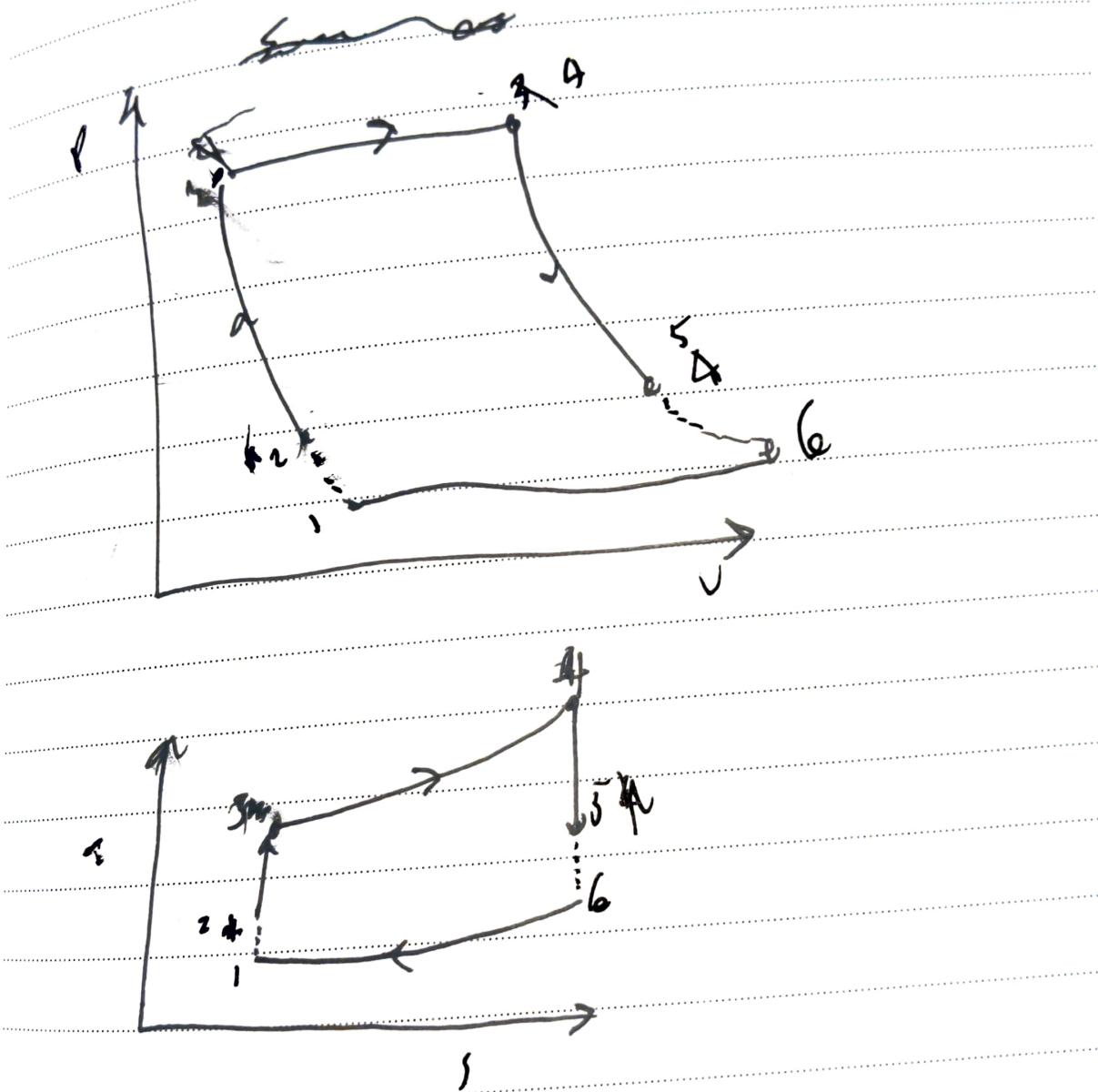
4-1 Gas change without rejection

c) Brumfitt Cycle



- Assumption:
1. "Glycoly" system with no heat loss to surroundings
 2. The air is treated as an ideal gas
 3. ΔT_{in} & ΔT_{out} are negligible
 4. The heat transfer is adiabatic
- These assumptions are valid for the Brumfitt cycle.

d) Jet - Rankine cycle



Ansatz: Standard ent. assumptions

553 K

Area flow constants

α2 für den Ottozyklus

$$\eta_{\text{Otto}} = 1 - \frac{T_4 - 1}{T_3 - T_2}$$

$$\frac{T_4}{T_2} = \left(\frac{V_2}{V_1}\right)^{k-1} = \left(\frac{V_3}{V_4}\right)^{k-1} = \frac{T_4}{T_3}$$

$$\frac{V_1}{V_2} = \frac{V_{\text{max}}}{V_{\text{min}}} = \frac{V_1}{V_2} = \frac{V_1}{V_2}$$

ausgewählt

Flammmelänge 90 mm und 40°C

$$pV = RT$$

$$V = \frac{RT}{P}$$

6000 rpm 90 kW acht

$$\eta_{\text{Otto}} = 1 - \frac{1}{r^{n-1}}$$

$$= 1 - \frac{1}{1.4 - 1}$$

$$10.5 \cdot 0.95$$

$$= 0.68 \text{ Adm}$$

$$\eta_{\text{Mech}} = \frac{W_{\text{net}}}{a_{\text{in}}}$$

$$a_{\text{in}} = \frac{W_{\text{net}}}{0.6095}$$

$$\approx 147.66 \text{ kW}$$

$$C3 \approx 18.7$$

now add loss and pass 77% @ 100 kPa

$$c) r_c = \frac{V_3}{V_2}$$

$$\eta_{\text{th, ideal}} = 1 - \frac{1}{r^{n-1}} \left[\frac{r^n - 1}{R(r_c - 1)} \right]$$

$$V_2 = \frac{V_1}{r}$$

$$\text{for process 2-3} \frac{P_2}{P_1} = \frac{V_1}{V_2} = C \quad T_3 = T_{\text{max}}$$

$$\nu_1 = \frac{R_1}{P}$$

$$= 0.861 \frac{m^3}{kg}$$

$$r = \frac{v_2}{v_1} \frac{v_1}{v_2}$$

$$\nu_2 = \frac{0.861}{18.2}$$

$$= 0.0473 \frac{m^3}{kg}$$

$$P_2 = 100 \times 10^3 \left(\frac{v_1}{v_2} \right)^n$$

$$= 100 \times 10^3 \left(\frac{0.861}{0.0473} \right)^{1.4}$$

$$= 5.86 \text{ MPa}$$

$$T_2 = 300 \times \left(\frac{0.861}{0.0473} \right)^{0.4}$$

$$= 955.2 \text{ K}$$

$$PV = RT$$

$$\frac{P}{T} \propto \frac{V}{V}$$

$$PV = RT$$

$$\frac{P_1 V_1}{V_2} = \frac{P_2 V_2}{V_3}$$

$$\left(\frac{955.2}{0.0473} \right) \times \frac{1700}{100} = V_3$$

$$V_3 = 0.0842 \text{ m}^3/\text{kg}$$

$$r_2 = \frac{0.0842}{0.0473}$$

$$= 1.78$$

$$T_4 = T_3 \left(\frac{V_3}{V_4} \right)^{n-1}$$

$$= 1700 \left(\frac{0.0842}{0.861} \right)^{0.4}$$

$$= 670.7 \text{ K}$$

670.7 \rightarrow 300

For Process 4-1

work = 0 as $dv = 0$

$\therefore \Delta U = 0$

$u = Cp \Delta T$

$$= 1005 \times -370.7$$

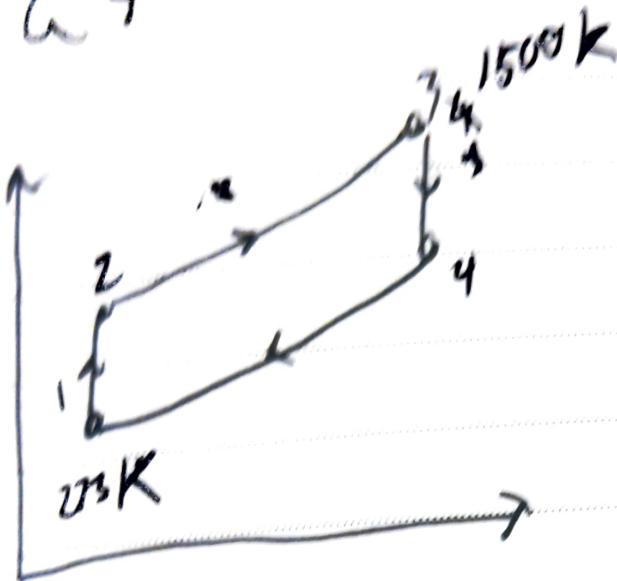
$$= -372.5 \text{ kJ}$$

$\therefore -372.5 \text{ kJ}$ is rejected from the system

$$Q) \eta_{\text{Reversible}} = 1 - \frac{1}{18.2} \left(\frac{1.74^{0.4} - 1}{1.4(1.74 - 1)} \right)$$

$$= 0.645$$

a 4



$$P_2 = \sqrt{P_1} \\ = 700 \text{ kPa}$$

$$T_2 = T_1 \left(\frac{P_2}{P_1} \right)^{\frac{\alpha}{1.4}}$$

$$= 273 \left(\frac{700}{100} \right)^{\frac{0.4}{1.4}}$$

$$= 476.0 \text{ K}$$

$$\frac{\rho}{\bar{\rho}} = \frac{T}{T_0}$$

$$\frac{T_2}{T_1} = \frac{T_3}{T_2}$$

$$V_2 = \frac{P R T}{P}$$

$$= \frac{297.476}{1000}$$

$$= 0.0195$$

$$V_3 = \frac{V_2}{T_3}$$

$$V_3 = \frac{T_3 V_2}{T_1}$$

$$= \frac{1500 \cdot 0.0195}{476}$$

$$= 0.0614$$

$$\Delta T = 102.4$$

$$W_{\text{rev}} = \int v dP$$

$$\therefore W_{\text{rev}} = 0 \text{ as } dP = 0$$

$$Q_{\text{rev}} = m(\Delta h + \Delta h_e + \Delta p_e)$$

$$\Delta h = m(\Delta h)$$

$$\Delta h = m \left(\frac{735 \cdot 10^3}{1} \right)$$

$$\frac{c}{m} = 735.2 \times 10^3 \text{ J/kg}$$

1 kJ/kg/s

$$\eta_{\text{Brayton}} = 1 - \frac{1}{T^{\frac{c-4}{k-1}}}$$

$$\approx 0.426$$

$$0.426 \times 150 \times 10^6$$

$$\eta = \frac{\text{what is}}{\text{what is}} \frac{c}{c}$$

$$\eta = \frac{0.426 \times 735 \times 10^3}{150 \times 10^6} = m$$

$$m =$$

for expt &
Chapter 7.3 - 7.7

$$T = -\frac{1}{\rho} \left(\frac{\partial p}{\partial v} \right) T$$

$$T_p = -\frac{1}{\rho} \left(\frac{\partial p}{\partial v} \right)_T$$

$$T_s = -\frac{1}{\rho} \left(\frac{\partial p}{\partial v} \right)_s$$

when $M > 0.3$ flow is Mach
so considered compressible

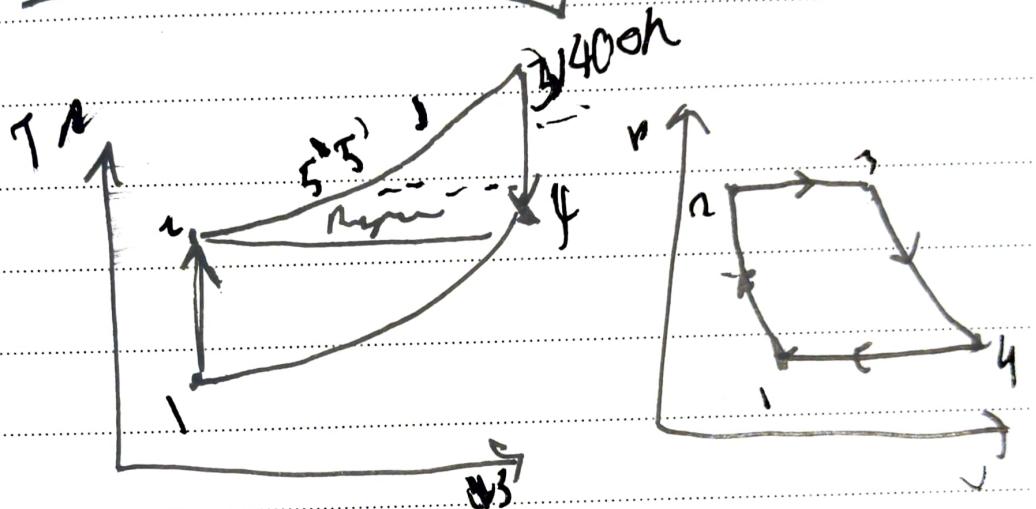
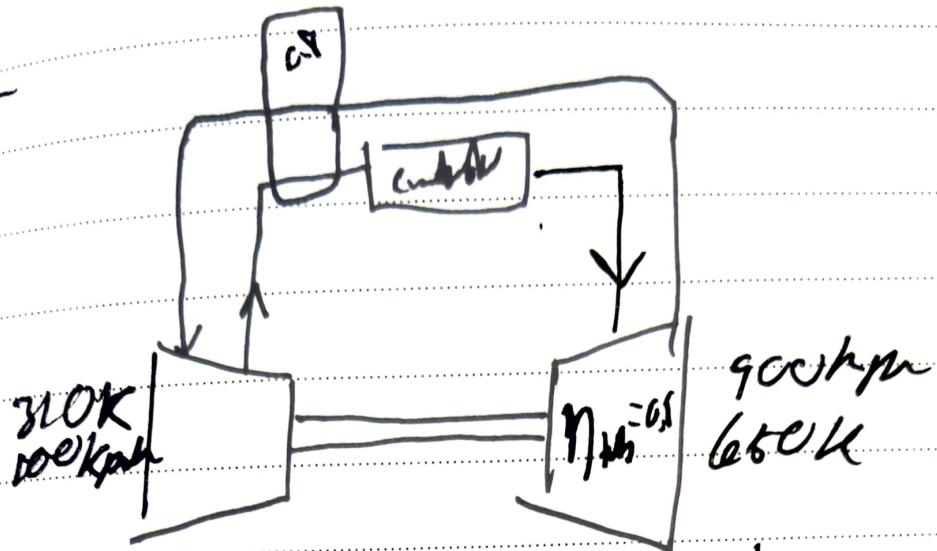
Assume adiabatic and inviscid
 $T_{\infty} = 0$

$$\frac{D \left(e + \frac{v^2}{2} \right)}{dt} = -\nabla \cdot \rho V$$

in adiabatic flow $T_1 = T_2$
 $h_1 = h_2$ ~~81~~

for isentropic flow $\rho_1 = \rho_2$ &

6.5



$$\epsilon = \frac{\text{Gross Cet}}{\text{Gross Wre}} = \frac{h_5^* - h_2^*}{h_4 - h_2}$$

$$v = \frac{RT}{P}$$

if add to

$$\text{state 1} = 310K, 100hPa, 0.8897 \frac{m^3}{kg}$$

$$\text{state 4} = 650K, 900hPa, 0.2072 \frac{m^3}{kg}$$

$$\text{state 3} = 1400K, 1120.6hPa, 0.3686 \frac{m^3}{kg}$$

$$\text{state 4} = 100hPa$$

$$V_3 P_3 = V_4 P_4$$

$$\text{state 2} = 618.3K, 120.6hPa, 0.158 \frac{m^3}{kg}$$

$$q_{\text{regen max}} = h_4 - h_2$$

$$h_4 = q_p \alpha T$$

$$= 1005 \times 650$$

$$h_3 = 1407000$$

$$h_4 \approx 311550$$

$$h = 653260 \text{ kJ}$$

$$h_2 = 621391$$

$$q_{\text{regen max}} = 653260 - 621391$$

$$= 38591 \text{ J}$$

$$q_{\text{regen actual}} = \epsilon \cdot q_{\text{regen max}}$$

$$= 25487.2 \text{ J}$$

↑

u) The Rankine cycle has 25487.2 J of heat input to the condenser

$$h_5 = q_{\text{regen max}} + h_2$$

$$= 646887.2$$

$$\text{am} = 760,122 \text{ kJ}$$

Ward

$$\dot{e} - \dot{w} = \dot{m} (Dh + Dke + Dke)$$

$$- \dot{m} =$$

$$w = Dh$$

$$w_{\text{ad}} = h_3 - h_2 \\ = 753750 \text{ J} \times$$

$$w_{\text{per}} = -(w_1 - w_2) \\ = 30984 \text{ J}$$

$$w_{\text{ad+per}} = 0.9 \times 753750 \text{ J} \\ = 838375 \text{ J}$$

$$w_{\text{arbeit}} = 5485345$$

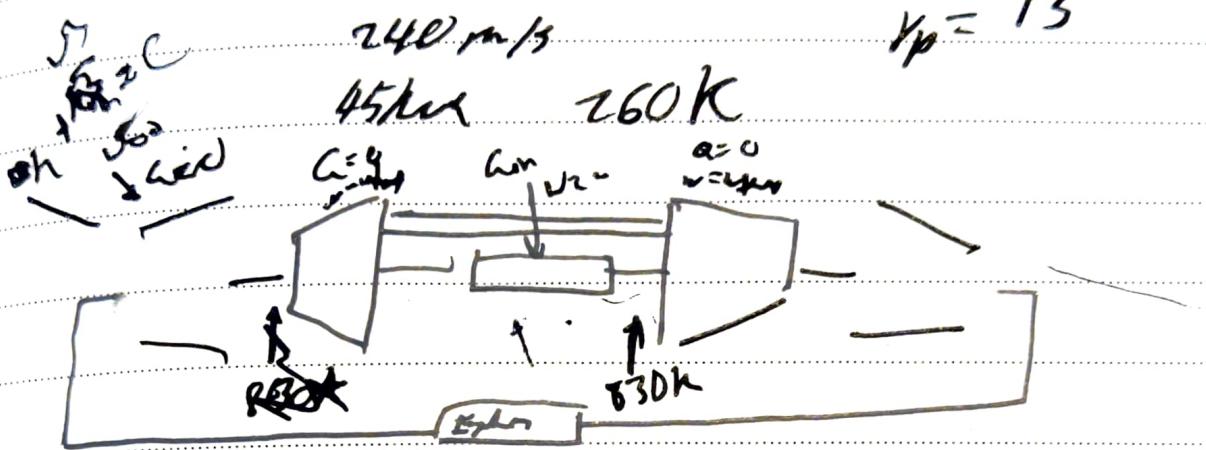
$$\eta_{\text{arbeit}} = \frac{5485345}{760122}$$

$$= 6.72$$

$$= 72\%$$

$$A = 7.01 \text{ m}^2$$

$$\nu_p = 13$$



$$v = \frac{RT}{P}$$

RTD Resistance
at 230 K

$$= 1.658 \frac{\text{m}^3}{\text{kg}}$$

$$\frac{P_2}{P_1} = \frac{\text{constant}}{P_2 = \frac{P_2}{P_1} = \frac{P_2}{P_1}}$$

$$\dot{V} = \bar{V} \cdot A$$

$$= 482.4 \text{ m}^3/\text{s}$$

$$m = \frac{1}{\rho} \dot{V}$$

$$= 290.95 \text{ kg/s}$$

$$w_{\text{friction}} = w_{\text{viscous}}$$

$$h_{\text{no drag}} = 134750$$

$$PV^{\beta} = C$$

$$P_m = \beta P_{atm}$$

$$\frac{V_{atm}}{V_m} = \frac{\beta P_{atm}}{P_m}$$

$$\frac{V_{atm}}{\beta} = V_m$$

$$P_{atm} = \frac{RT}{P_m V_m}$$

$$\begin{aligned} C &= P_m V_m + P_{atm} V_m^{\beta} \\ &= \beta P_{atm} V_m^{\beta} \\ &= \beta P_{atm} \left(\frac{V_{atm}}{\beta} \right)^{\beta} - P_{atm} \end{aligned}$$