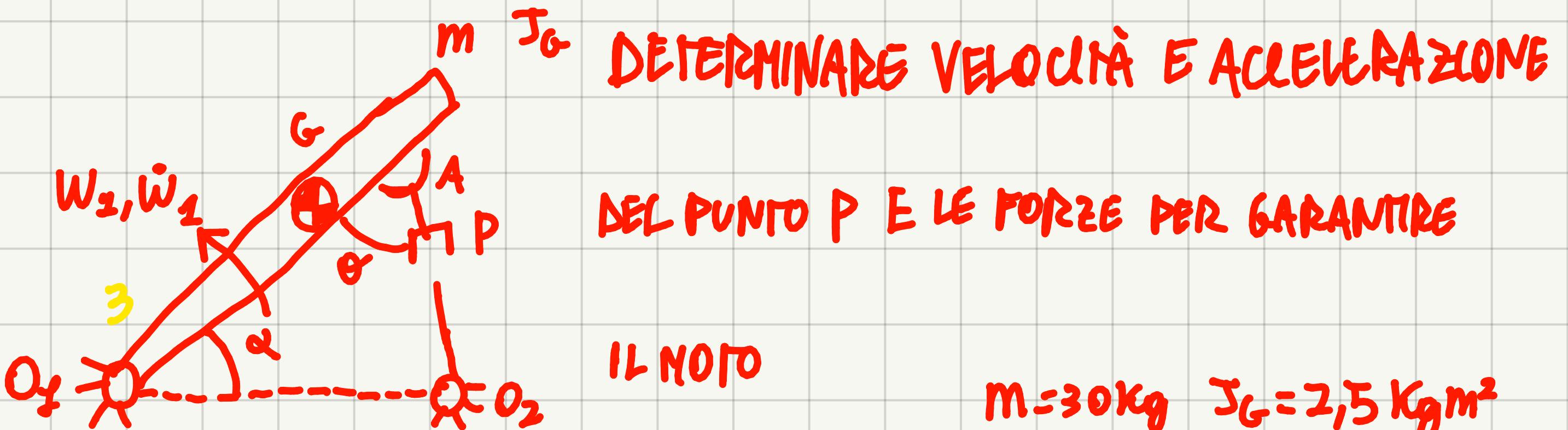


# DINAMICA

① CONSIDERARE IL SEGUENTE SISTEMA. CON LE INFORMAZIONI NOTE,

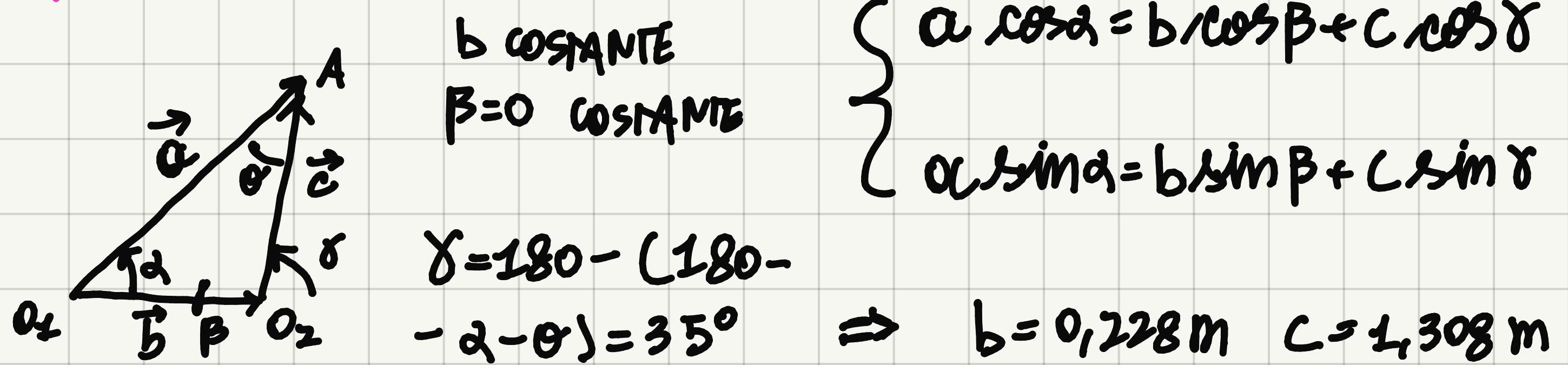


$$m = 30 \text{ kg} \quad J_G = 2,5 \text{ kg m}^2$$

$$\overline{O_1G} = 1 \text{ m} \quad \overline{O_1A} = 1,5 \text{ m} \quad \theta = 5^\circ \quad \alpha = 30^\circ \quad \omega_1 = 0,5 \text{ rad/s} \quad \dot{\omega}_1 = 0,5 \text{ rad/s}^2$$

$$n = 3 \cdot 3 - (2\omega_1 + 2\alpha_1 + 2\dot{\omega}_1) = 1$$

$\vec{v}_P$  E  $\vec{\alpha}_P$  CON LA CINEMATICA



$$\left\{ \begin{array}{l} -a \ddot{\alpha} \sin \alpha = c \sin \gamma - c \dot{\gamma} \sin \alpha \\ a \ddot{\alpha} \cos \alpha = c \sin \alpha + c \dot{\gamma} \cos \alpha \end{array} \right.$$

$$\begin{vmatrix} \cos \alpha & -c \sin \alpha \\ \sin \alpha & c \cos \alpha \end{vmatrix} \cdot \begin{vmatrix} \dot{\gamma} \\ \ddot{\gamma} \end{vmatrix} = \begin{vmatrix} -a \ddot{\alpha} \sin \alpha \\ a \ddot{\alpha} \cos \alpha \end{vmatrix} \quad \det A = c$$

$$\dot{\gamma} = \frac{1}{c} \cdot \det \begin{vmatrix} -a \ddot{\alpha} \sin \alpha & -c \sin \alpha \\ a \ddot{\alpha} \cos \alpha & c \cos \alpha \end{vmatrix} = 0,065 \text{ rad/s}$$

$$\ddot{\gamma} = \frac{1}{C} \cdot \det \begin{vmatrix} \cos\delta & -\alpha_2 \sin\delta \\ \sin\delta & \alpha_2 \cos\delta \end{vmatrix} = 0,5712 \text{ rad/s}$$

$$C = |\vec{v}_p|$$

$$\left\{ \begin{array}{l} -\alpha_2 \ddot{\gamma} \sin\delta - \alpha_2^2 \cos\delta = \ddot{C} \cos\delta - \dot{C} \dot{\gamma} \sin\delta - \dot{C} \dot{\gamma} \sin\delta - C \ddot{\gamma} \sin\delta - C \dot{\gamma}^2 \cos\delta \\ \alpha_2 \ddot{\gamma} \cos\delta - \alpha_2^2 \sin\delta = \ddot{C} \sin\delta + \dot{C} \dot{\gamma} \cos\delta + \dot{C} \dot{\gamma} \cos\delta + C \ddot{\gamma} \cos\delta - C \dot{\gamma}^2 \sin\delta \end{array} \right.$$

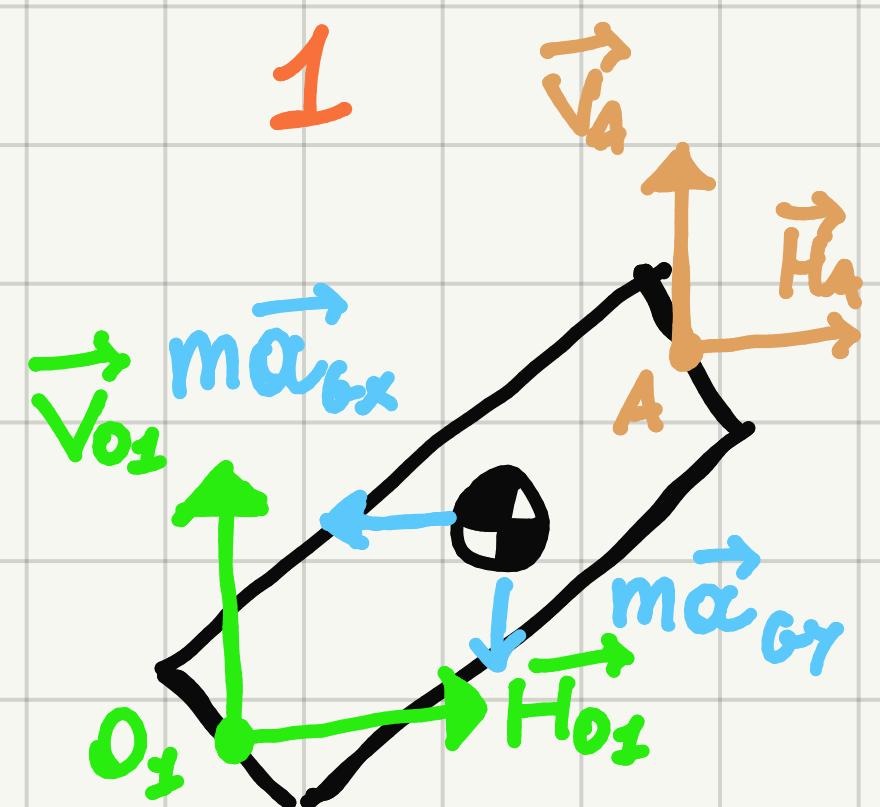
DA NOTARE I TERMINI CHE SI SOMMANO. SE SI NOTA SI ANNULLANO, CI SONO  
STATI ERRORE DI CALCOLO

$$\rightarrow \left\{ \begin{array}{l} \ddot{C} = 0,119 \text{ m/s}^2 \\ \ddot{\gamma} = 0,54 \text{ rad/s}^2 \end{array} \right. \quad C = |\vec{v}_p|$$

## DINAMICA ( D'ALEMBERT )

TENERE IN CONSIDERAZIONE REAZIONI VINCOLARI, FORZE ESTERNE E  
FORZE D'INERZIA ( $\vec{F}_{in} = -m\vec{a}$ ;  $\vec{C}_{in} = -J_G \vec{\omega}$ )

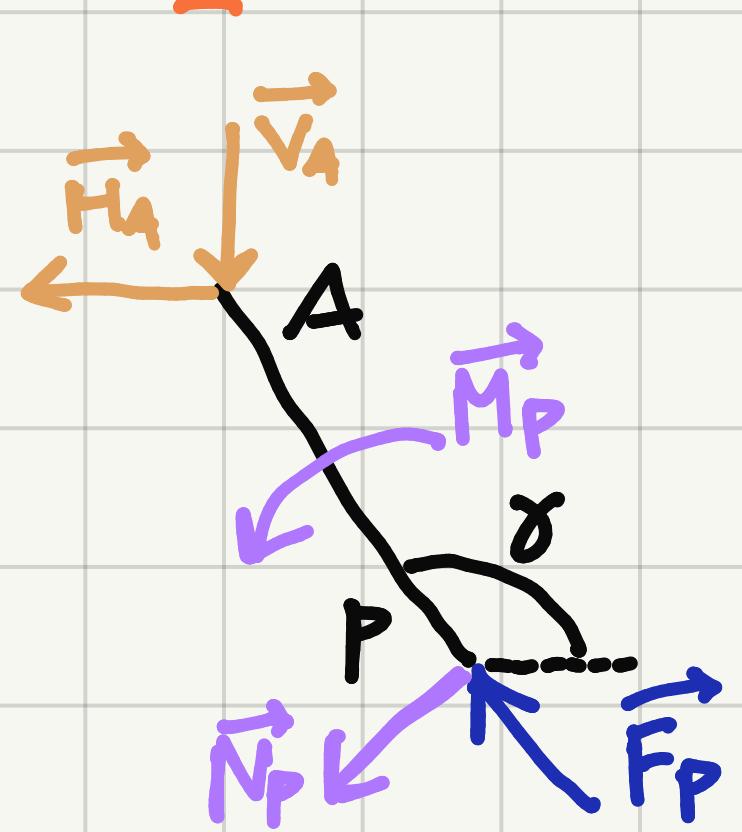
SUCCESSIONALMENTE, STUDIARE I SISTEMI DI FORZE PER OGNI CORPO



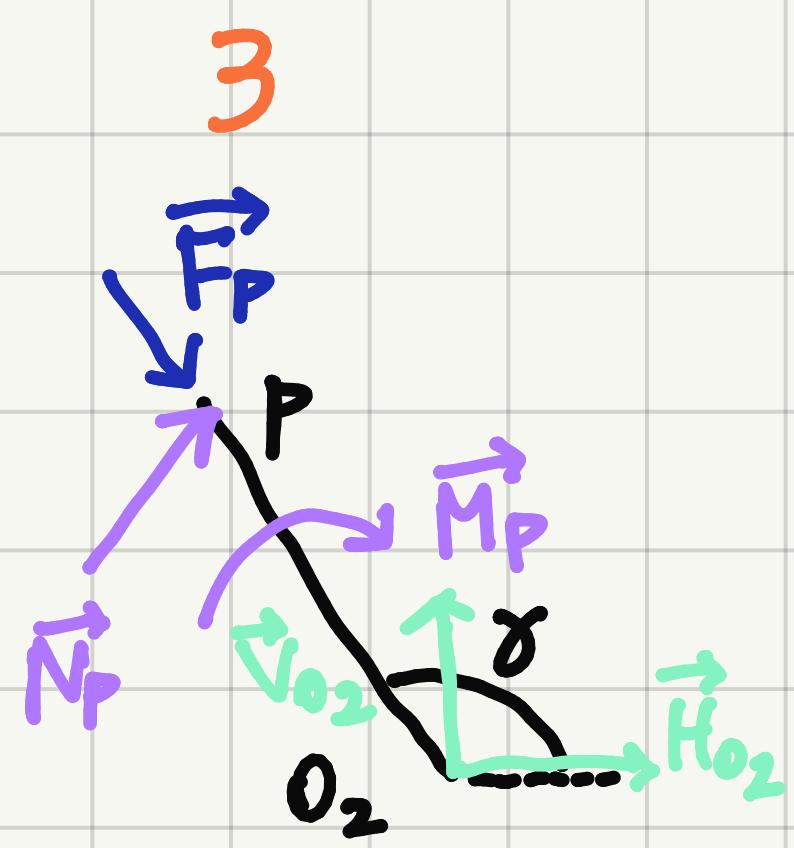
NEL BARICENTRO, SUPPORRE FORZE COI VERSI  
OPPOSTI ALLE CONVENZIONI

$$\times \left\{ \begin{array}{l} H_{0x} + H_A - m a_{gx} = 0 \\ V_{0x} + v_A - m a_{gy} = 0 \end{array} \right.$$

$$M_{0x} \left\{ \begin{array}{l} -J_G \dot{\omega}_y + m a_{gy} \cdot \overline{O_x A} \cos(\pi - \alpha) + \\ + m a_{gx} \cdot \overline{O_x A} \sin\alpha + V_A \cdot \overline{O_x A} \cos\alpha - H_A \cdot \overline{O_x A} \sin\alpha = 0 \end{array} \right.$$



$$\left\{ \begin{array}{l} -H_A + F_p \cos\gamma - N_p \sin\gamma = 0 \\ -V_A + F_p \sin\gamma + N_p \cos\gamma = 0 \\ M_p - N_p A_P = 0 \end{array} \right. \quad ! \text{ IPOTESI } \gamma \leq \pi/2$$



$$\begin{cases} H_{O_2} - F_P \cos \delta + N_P \sin \delta = 0 \\ V_{O_2} - F_P \sin \delta - N_P \cos \delta = 0 \\ -M_P - N_P \overline{O_2 P} = 0 \end{cases}$$

OTTIENIAMO UN SISTEMA FINALE DI 9 EQUAZIONI IN 9 INCOGNITE

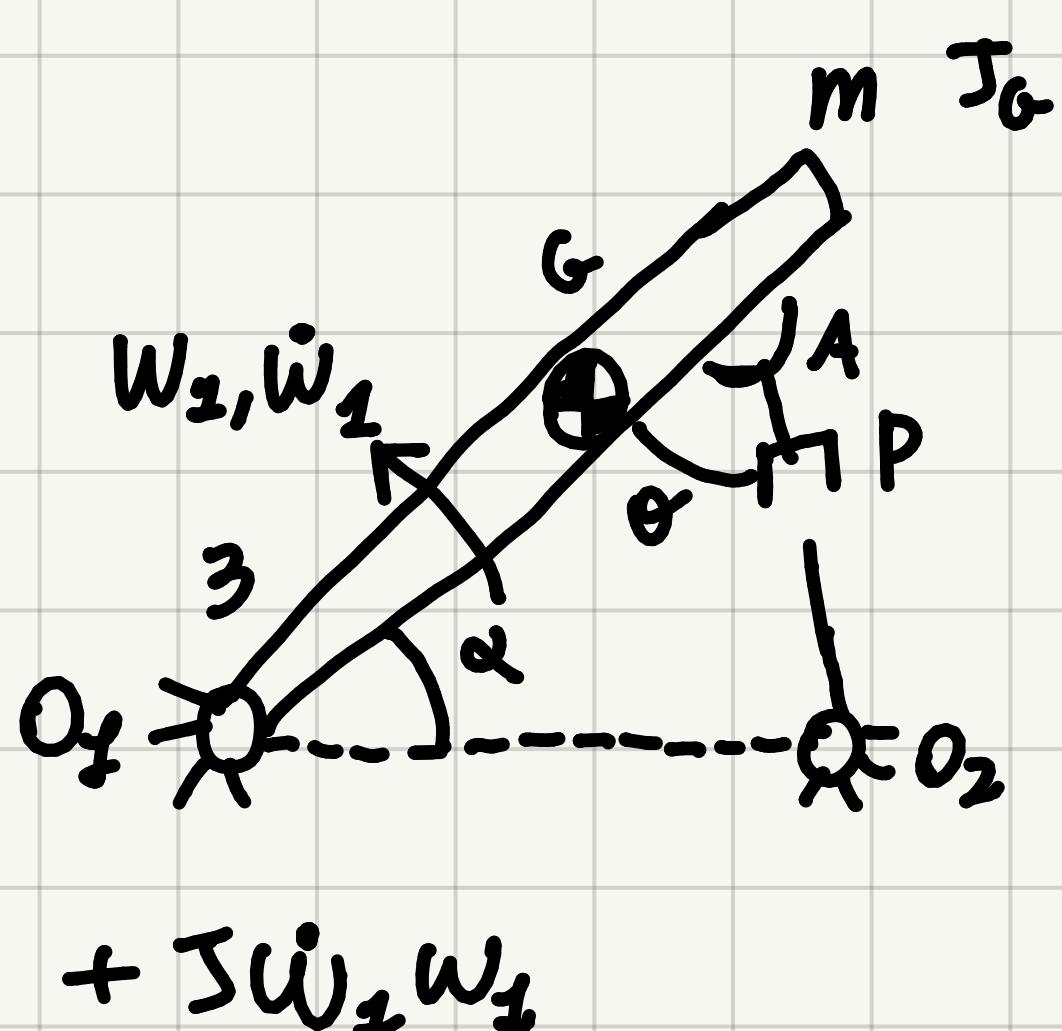
INIZIANO OSSERVANO CHE, ESSENDO P ANCORATA A TERRA,  $M_P = N_P = 0$

DA 2  $\begin{cases} F_P \cos \delta = H_A \\ F_P \sin \delta = V_A \end{cases}$   $\delta = 35^\circ$ , COME VISTO IN PRECEDENZA

$$\Rightarrow J_G(\ddot{\omega}_z - m a_{Gx} \overline{O_2 A} \cos \delta + m a_{Gy} \overline{O_2 A} \sin \delta + F_P \overline{O_2 A} \sin \delta \cos \delta -$$

$$- F_P \overline{O_2 A} \cos \delta \sin \delta) = 0 \Rightarrow F_P = 124,23 \text{ N}$$

## DINAMICA (ENERGIA)



$$\begin{aligned} \frac{d}{dt} K &= m (\vec{a}_G \cdot \vec{v}_G) + J_G (\vec{\omega}_z \cdot \vec{\omega}_z) \\ &= m (a_{Gx} \hat{i} + a_{Gy} \hat{j}) \cdot (V_{Gx} \hat{i} + V_{Gy} \hat{j}) + \\ &+ J_G (\dot{\omega}_z \hat{k}) \cdot (\omega_z \hat{k}) = m a_{Gx} V_{Gx} + m a_{Gy} V_{Gy} + \end{aligned}$$

$$+ J \ddot{\omega}_z \omega_z$$

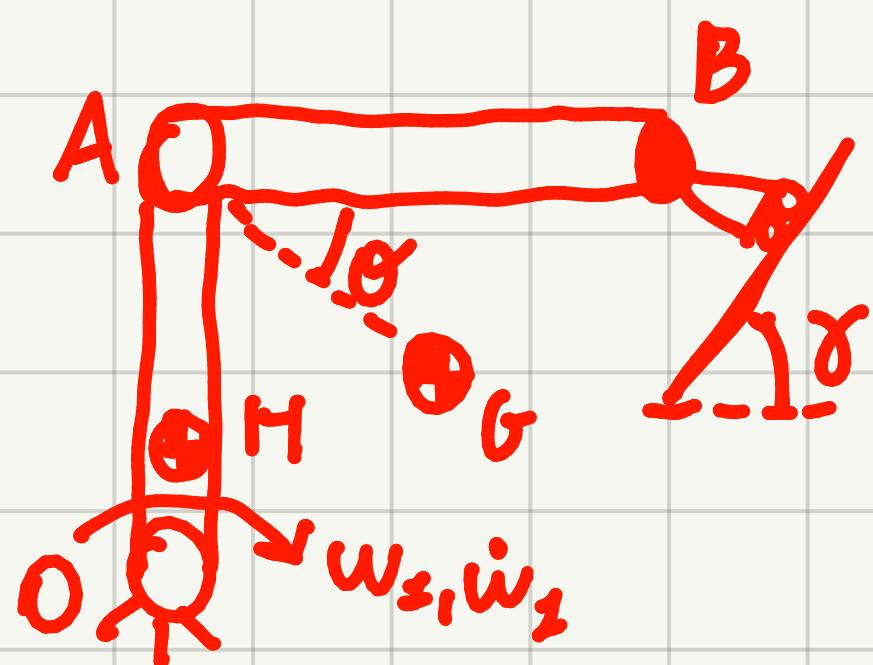
$$\sum P_{\text{ATTIVE}} = \vec{F}_P \cdot \vec{v}_P = F_P v_P$$

! I VINCOLI, AVENDO VELOCITÀ NUDA, NON GENERANO POTENZA

$$\sum P_{\text{ATTIVE}} = \frac{d}{dt} K \rightarrow F_P v_P = m a_{Gx} V_{Gx} + m a_{Gy} V_{Gy} + J_G \dot{\omega}_z \omega_z$$

$$\Rightarrow F_P = 124,23 \text{ N}$$

② CONSIDERARE IL SEGUENTE SISTEMA. CON LE INFORMAZIONI NOTE,



DETERMINARE VELOCITÀ E ACCELERAZIONE DEL

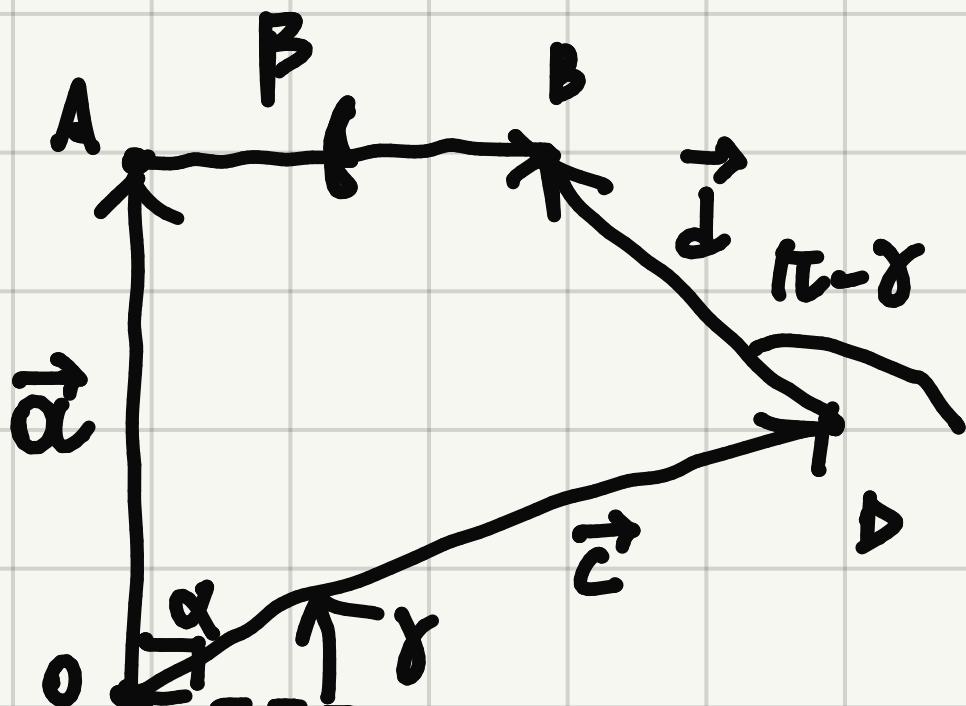
PUNTO B E LA COPPIA IN OA PER GARANTIRE IL MOTORE

$$OA = 0,45 \text{ m} \quad AB = 0,43 \text{ m} \quad AG = 0,1 \text{ m} \quad OH = 0,15 \text{ m} \quad w_z = 0,5 \text{ rad/s}$$

$$\dot{w}_z = 1,2 \text{ rad/s}^2 \quad M_B = 40 \text{ kg} \quad M_{OA} = 10 \text{ kg} \quad J_{OA} = 0,3 \text{ kg} \cdot \text{m}^2 \quad J_{AB} = 0,25 \text{ kg} \cdot \text{m}^2$$

$$n = 3 \cdot 2 - (2_0 + 2_A + 1_B) = 1$$

CINEMATICA



$\alpha, b, d, \gamma$  FISSI

$c, d, \beta$  NON FISSI

RISOLVENDO, SI OTTENGONO I RISULTATI SEGUENTI

$$\left\{ \begin{array}{l} \dot{\beta} = 0,3 \text{ rad/s} \\ \dot{c} = 0,26 \text{ m/s} \end{array} \right.$$

$$|\vec{v}_B| = \dot{c}$$

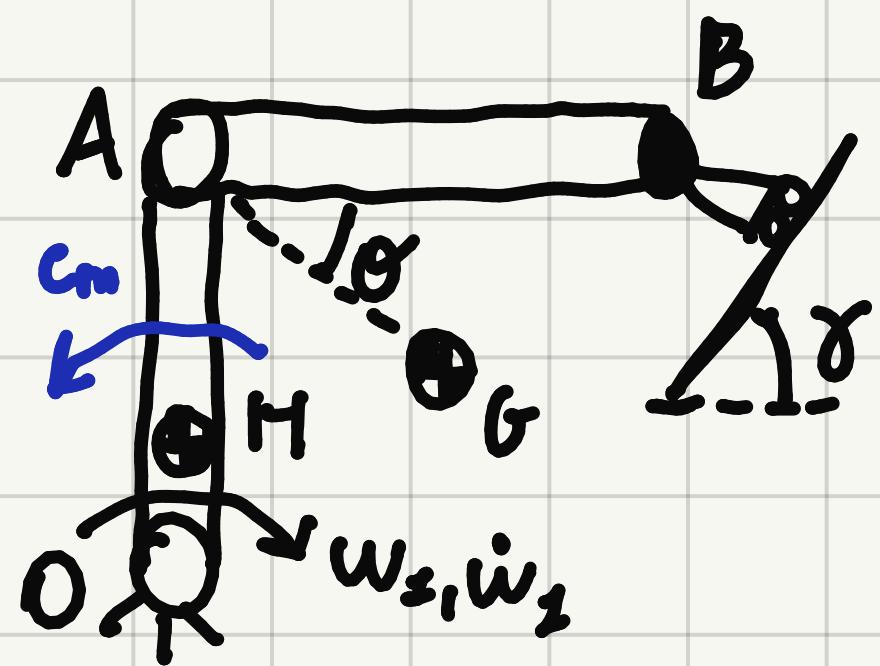
$$\left\{ \begin{array}{l} \ddot{\beta} = 0,93 \text{ rad/s}^2 \\ \ddot{c} = 0,58 \text{ m/s}^2 \end{array} \right.$$

$$|\vec{a}_B| = \ddot{c}$$

$$\vec{a}_G = \vec{a}_A + \vec{\omega}_{AB} \times (\vec{G} - \vec{A}) - \vec{\omega}_{AB}^2 (\vec{G} - \vec{A})$$

$$\vec{a}_A = \vec{a}_O + \vec{\omega} \times (\vec{A} - \vec{O}) - \vec{\omega}^2 (\vec{A} - \vec{O})$$

## DINAMICA (D'ALEMBERT)

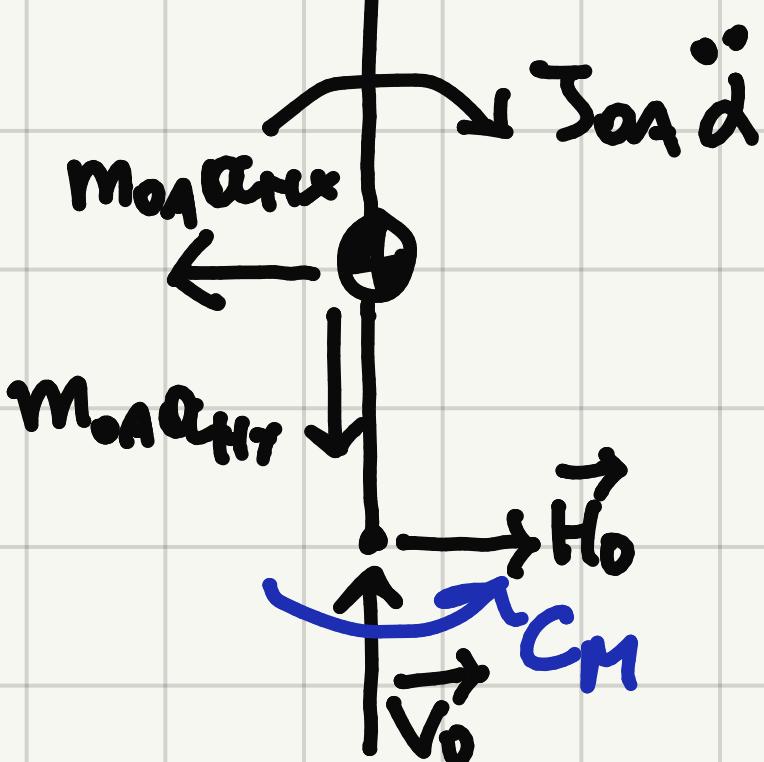


VOGUANO CHE L'ASTA OA LE QUINDI OB)  
NON CAMBI LA SUA INCLINAZIONE

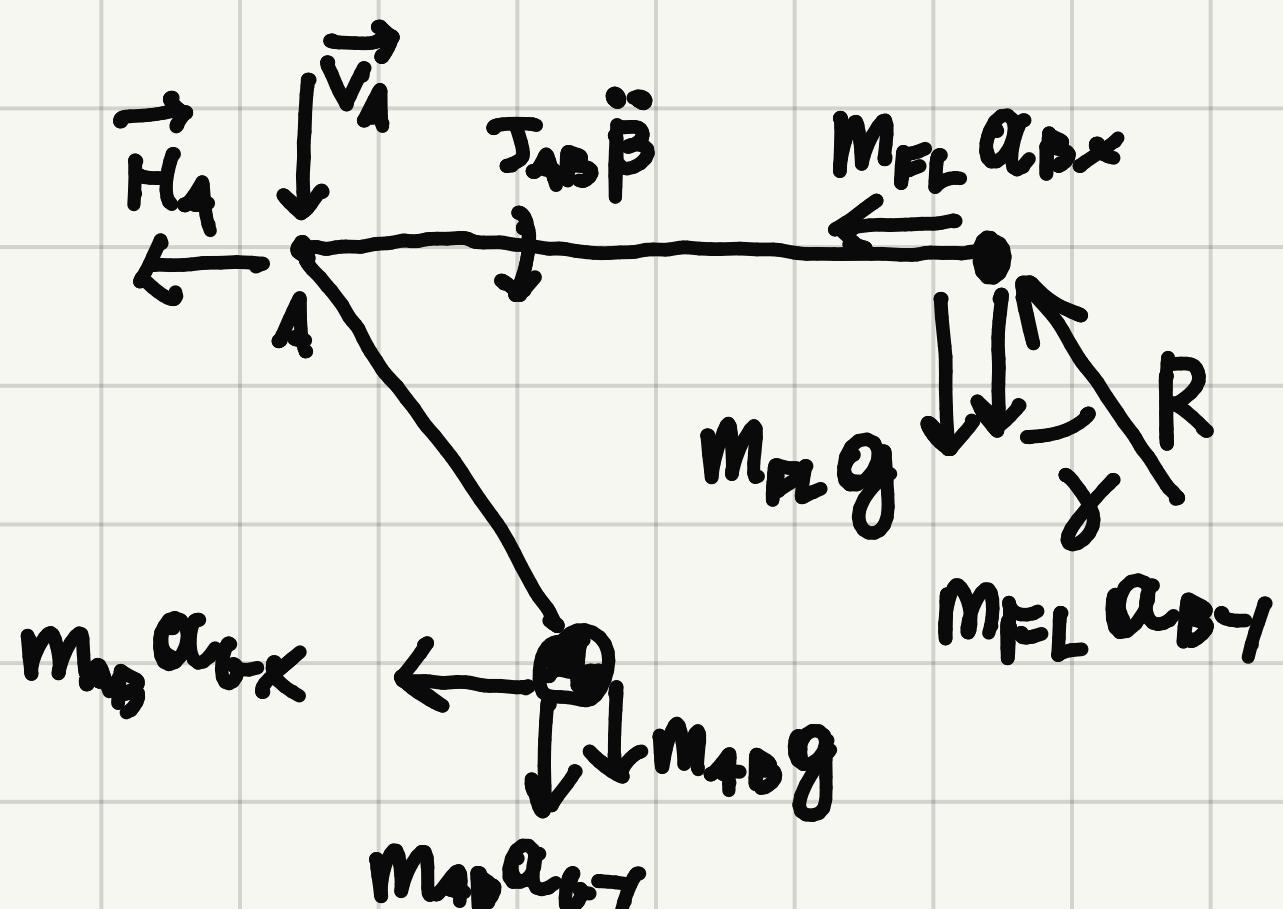
1

$$\begin{matrix} \vec{V}_A \\ \vec{H}_A \end{matrix}$$

$$\left\{ \begin{array}{l} H_A + H_0 - M_{OA} \alpha_{Hx} = 0 \\ V_0 + V_A - M_{OA} \alpha_{Hy} = 0 \\ \text{INO} \quad C_m - J_{Ox} \ddot{\theta} + M_{OA} \alpha_{Hx} \overline{OA} - H_A \overline{OA} = 0 \end{array} \right.$$



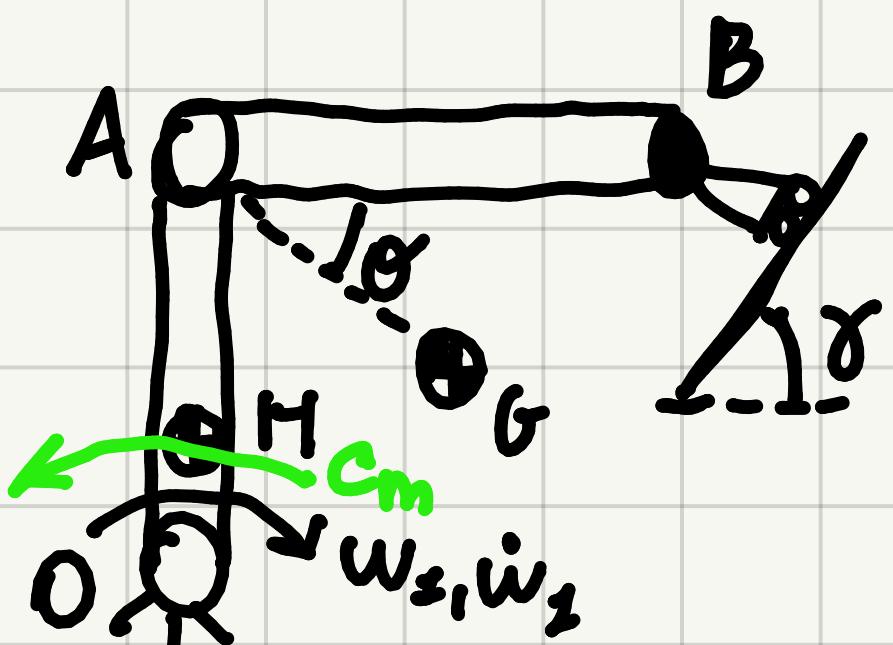
2



$$\left\{ \begin{array}{l} -H_A - R \sin \gamma - M_{AB} \alpha_{Bx} - M_{FL} \alpha_{Bx} = 0 \\ -V_A + R \cos \gamma - M_{AB}(g + \alpha_{Ay}) - M_{FL}(g + \alpha_{By}) = 0 \\ -J_{AB} \ddot{\beta} - M_{AB} \alpha_{Bx} \overline{AB} \sin \theta - M_{AB} (\alpha_{By} + g) \overline{AB} \cos \theta - M_{FL} (g + \alpha_{By}) \overline{AB} + R \overline{AB} \cos \gamma = 0 \end{array} \right.$$

$$\Rightarrow C_M = -118,8 \text{ N}\cdot\text{m} \Rightarrow \vec{C}_M = -118,8 \hat{k} \text{ N}\cdot\text{m}$$

## DINAMICA (ENERGIA)



$$\frac{d}{dt} K = \frac{d}{dt} K^{COAS} + \frac{d}{dt} K^{(AB)} + \frac{d}{dt} K^{(B)}$$

$$\frac{d}{dt} K^{(AB)} = M_{OA} (\vec{\alpha}_H \cdot \vec{v}_H) + J_{OA} (\vec{\omega}_{OA} \cdot \vec{w}_{OA})$$

$$= M_{OA} \alpha_{HX} v_{HX} + M_{OA} \alpha_{HY} v_{HY} + J_{OA} \ddot{\alpha} \dot{\alpha} \quad \ddot{\alpha} = -\ddot{w}_z, \dot{\alpha} = -\dot{w}_z$$

$$\text{ANALOGAMENTE, } \frac{d}{dt} K^{(B)} = M_{AB} \alpha_{Bx} v_{Bx} + M_{AB} \alpha_{By} v_{By} + J_{OB} \ddot{\beta} \dot{\beta}$$

$$\frac{d}{dt} K^{(B)} = M_{FL} (\vec{\alpha}_B \cdot \vec{v}_B) + J (\vec{\omega}_B \cdot \vec{w}_B) \quad \begin{array}{l} \text{MASSA PUNIFORME} \\ \Rightarrow \text{NO ROTAZIONE} \end{array}$$

$$= M_{FL} \alpha_B v_B$$

$$\sum P_{\text{ATTIVI}} = P^{(OAS)} + P^{(AB)} + P^{(B)} = (M_{OA} \vec{g} \cdot \vec{v}_H + \vec{C}_m \cdot \vec{w}_{OA}) +$$

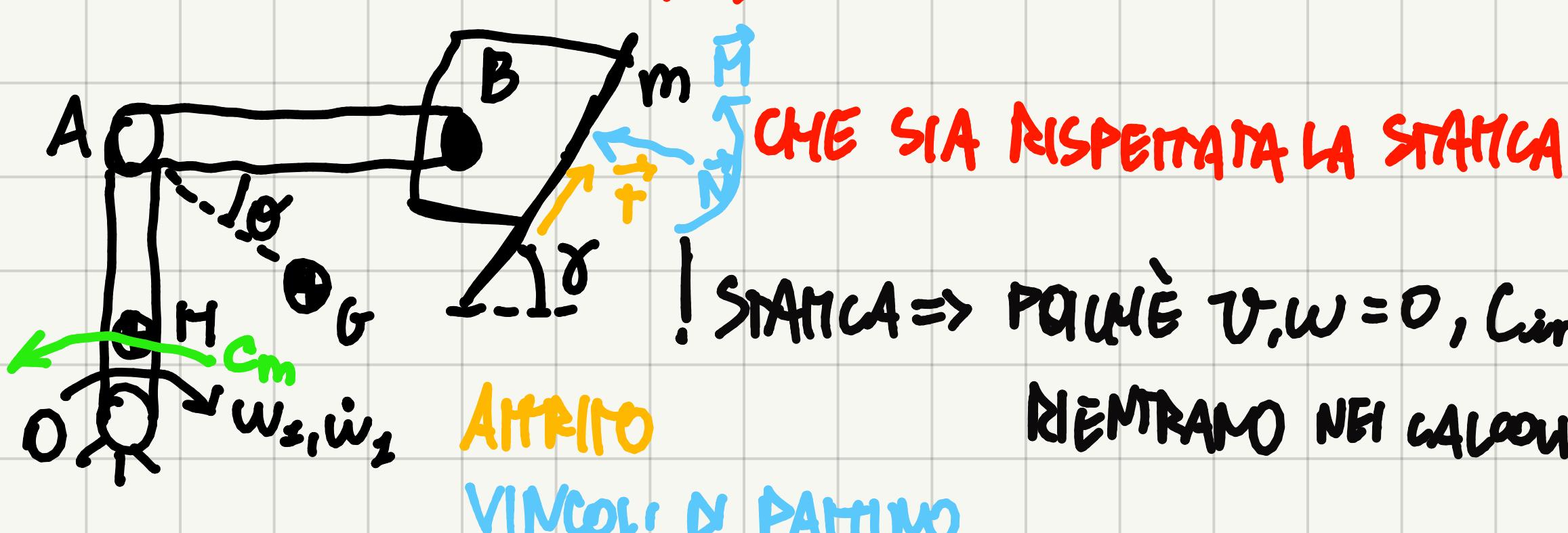
$$+ (M_{AB} \vec{g} \cdot \vec{v}_B) + (M_{FL} \vec{g} \cdot \vec{v}_B) = (M_{OA} (g \hat{z} \cdot (v_{HX} \hat{x} + v_{HY} \hat{y})) +$$

$$+ C_m \hat{K} \cdot \dot{\alpha} \hat{K}) + M_{AB} (g \hat{z} \cdot (v_{Bx} \hat{x} + v_{By} \hat{y})) + M_{FL} (g \hat{z} \cdot (v_{Bx} \hat{x} +$$

$$+ v_{By} \hat{y})) = -M_{OA} g v_{HY} - M_{AB} g v_{By} - M_{FL} g v_{By} + C_m \dot{\alpha}$$

$$\sum P_{\text{ATTIVI}} = \frac{d}{dt} K \rightarrow C_m = -118,8 \text{ N} \cdot \text{m} \Rightarrow \vec{C}_m = -118,8 \hat{K} \text{ N} \cdot \text{m}$$

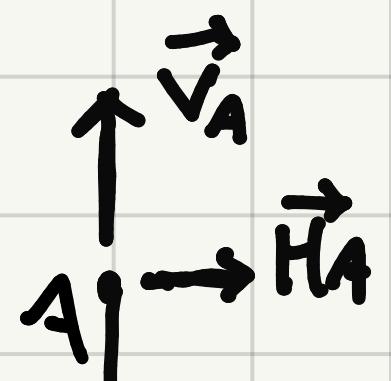
• PUNTO EXTRA: VERIFICA DI ADERENZA. SOSTITUIRE IL CARRELLO B CON UNA MASSA DI  $M_B$  (MASSA-PIANO) PER UN  $M = M_B$  E VERIFICARE



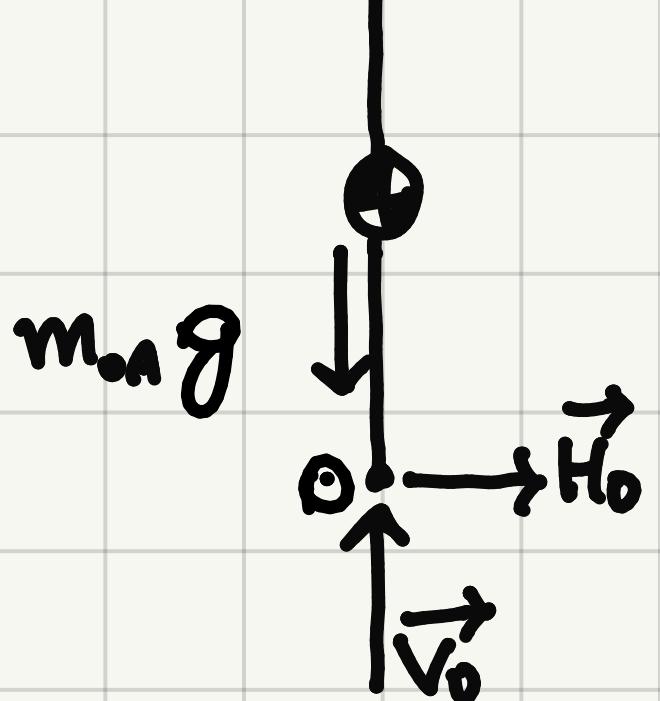
CHE SIA RISPECTATA LA STATICA

! STATICA  $\Rightarrow$  POURE V, W = 0, Cm È FIN NON RIENTRAMO NEI CALCOLI

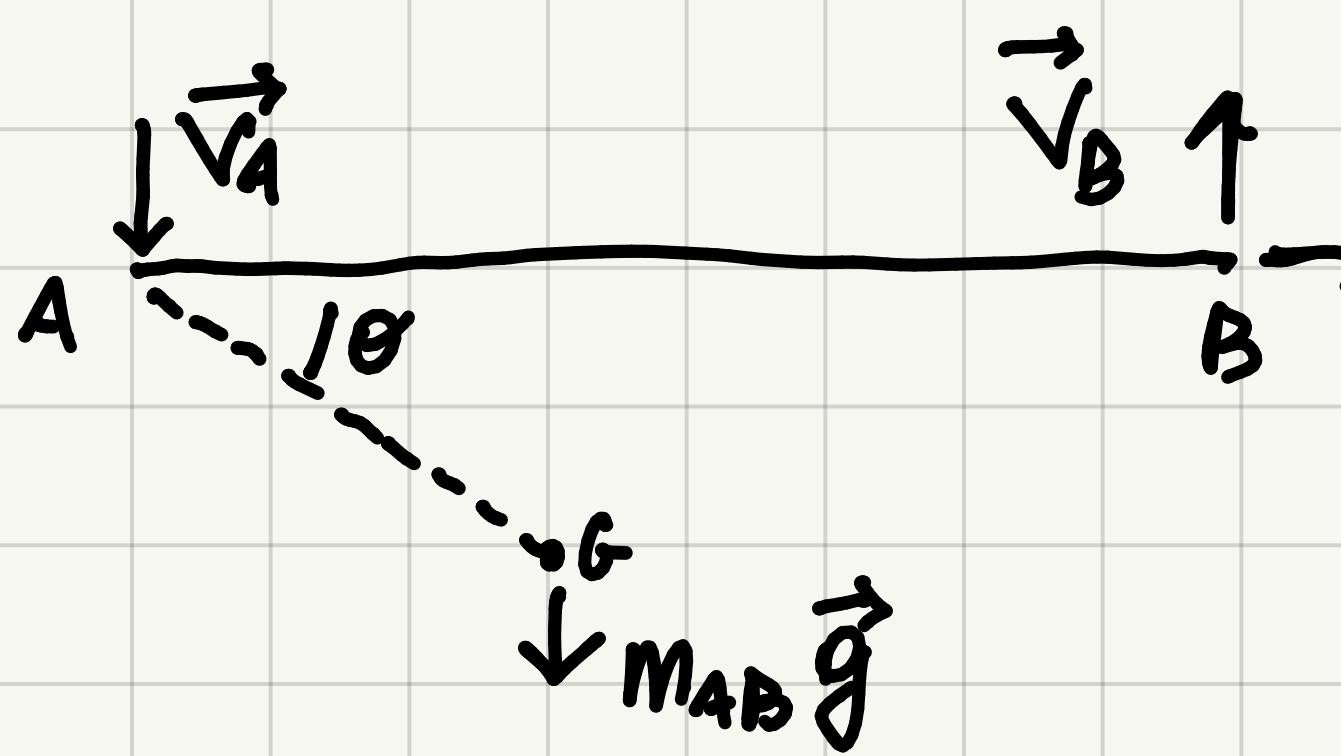
VINCOLI DI PARTECIPAZIONE



$$\left\{ \begin{array}{l} H_0 + H_A = 0 \\ V_A + V_0 - M_{OA} g = 0 \\ -H_A \cdot OA = 0 \end{array} \right. \Rightarrow H_A = 0$$

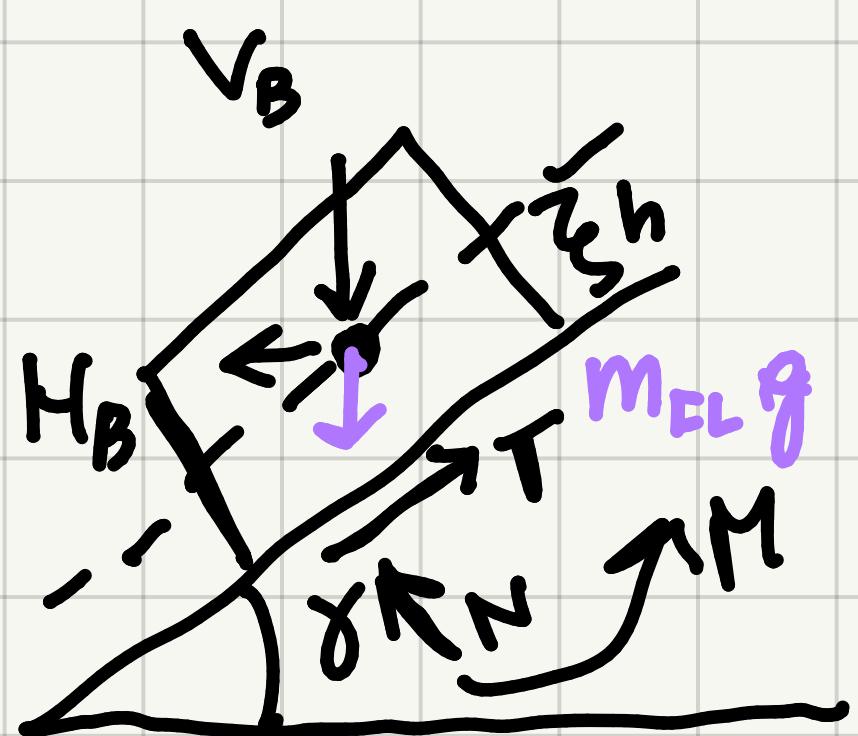


$$\left\{ \begin{array}{l} H_0 = 0 \\ V_1 + V_0 - M_{OA} g = 0 \\ H_A = 0 \end{array} \right.$$



$$\left\{ \begin{array}{l} H_B = 0 \\ -V_A + V_B - M_{AB} g = 0 \\ V_B \bar{A}\bar{B} - M_{AB} g \bar{A}\bar{G} \cdot \cos \theta = 0 \end{array} \right.$$

$$\left\{ \begin{array}{l} V_A = M_{AB} g \left( \frac{A\bar{b}}{AB} \cos \theta - 1 \right) \\ V_B = M_{AB} g \cdot \frac{A\bar{b}}{AB} \cos \theta \end{array} \right.$$



$$\left\{ \begin{array}{l} \sum F_{\parallel \gamma} = 0 \\ \sum F_{\perp \gamma} = 0 \\ \sum M_B = 0 \end{array} \right. \quad \left\{ \begin{array}{l} T - (M_{FL} g + V_B) \sin \gamma = 0 \\ N - (M_{FL} g + V_B) \cos \gamma = 0 \\ M + F_f l = 0 \end{array} \right.$$

IRRILEVANTE PER L'ESERCIZIO

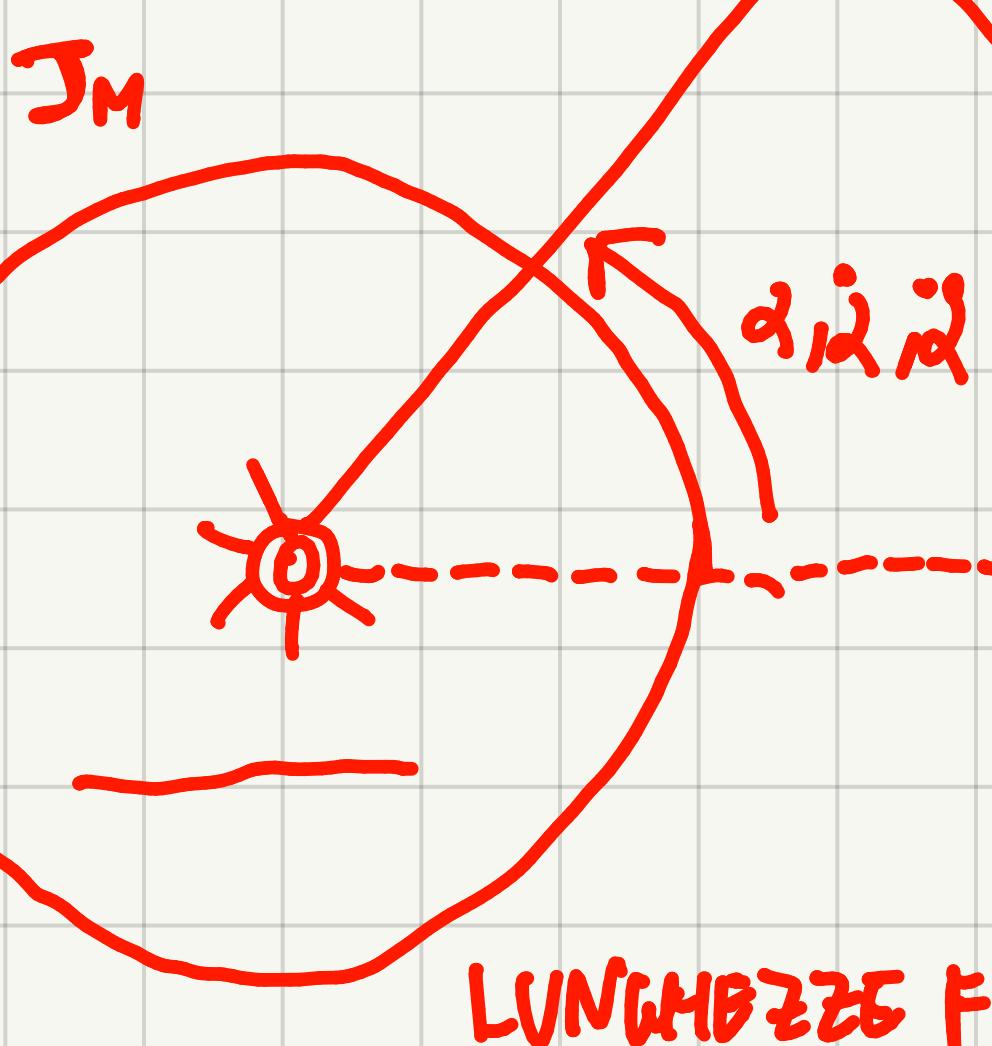
$$\left\{ \begin{array}{l} T = 201 N \\ N = 349 N \end{array} \right.$$

$$T_{\text{lim}} = N_s \cdot |\vec{N}| = 279$$

$T \leq T_{\text{lim}} \checkmark \Rightarrow$  ADERENZA VERIFICATA

CONSIDERIAMO UN MANOVELLISMO COMPOSTO DA UN DISCO SALDATO ALL'ASTA OA E

UNA SCATOLA IN B DI MASSA m. AL SISTEMA



È APPLICATA UNA COPPIA M\_R E LA

FORZA F. SIANO NOTI I

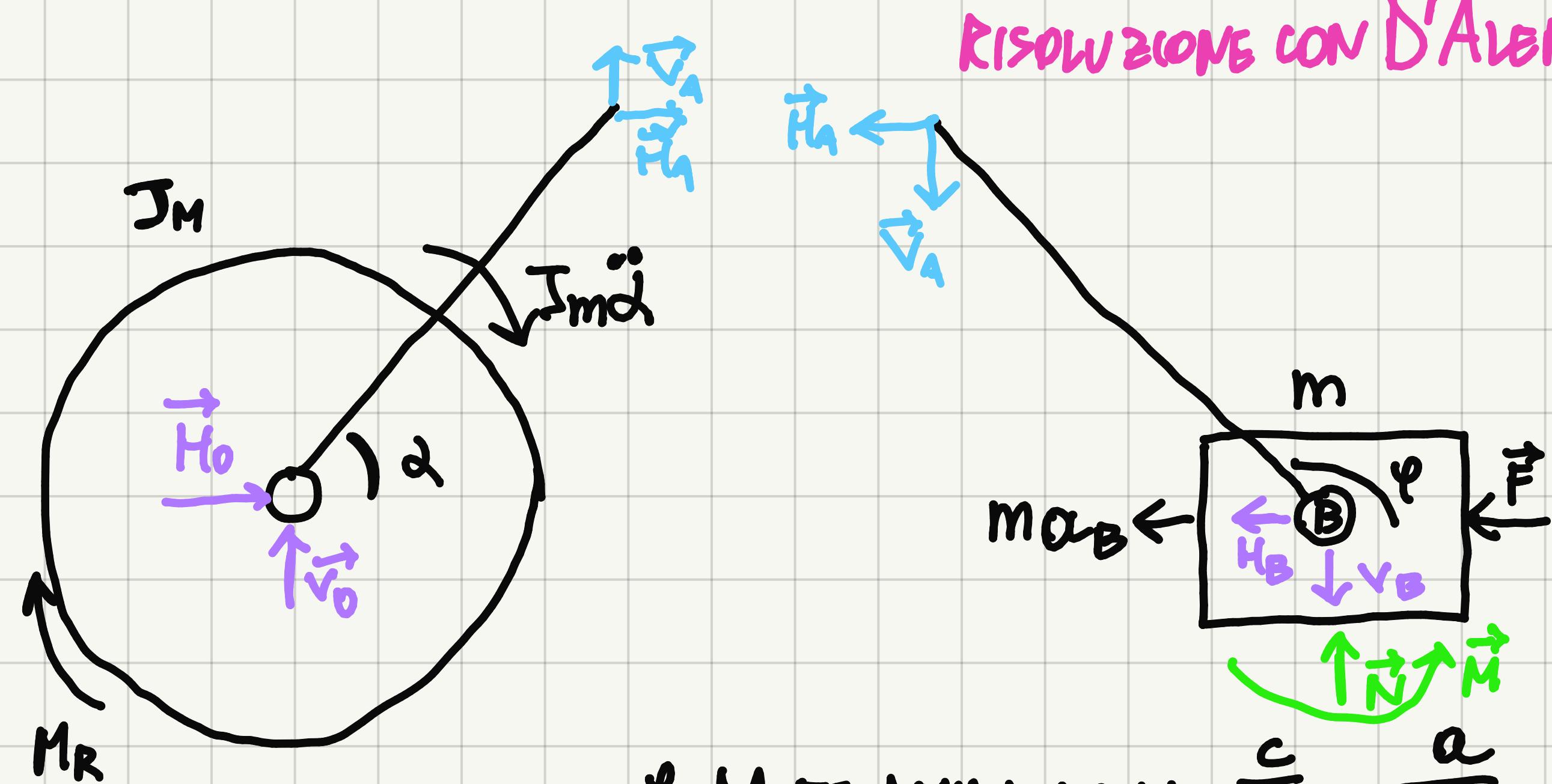
VALORI DI d, d, alpha, LE

LUNGHEZZE FISSE OA=a, OB=b, AB=c. SUPPONENDO

TRASCURABILI LE MASSE DELLE ASTE, CALCOLARE IL MOMENTO M\_R APPLICATO AL DISCO

IN MODO CHE VENGA GARANTITO IL MOTORE ASSEGNAZIO.

RISOLUZIONE CON D'ALEMBERT



DA TEOREMA DEL SEM  $\frac{c}{\sin \alpha} = \frac{a}{\sin(\pi - \varphi)}$

$$H_b + H_a = 0$$

$$V_a + V_b = 0$$

$$M_R + J_m \ddot{\alpha} + H_a a \sin \alpha - V_a a \cos \alpha \dot{\alpha} = 0$$

$$-H_a - H_b = 0$$

$$-V_a - V_b = 0$$

$$V_b b \cos(\pi - \varphi) + H_b b$$

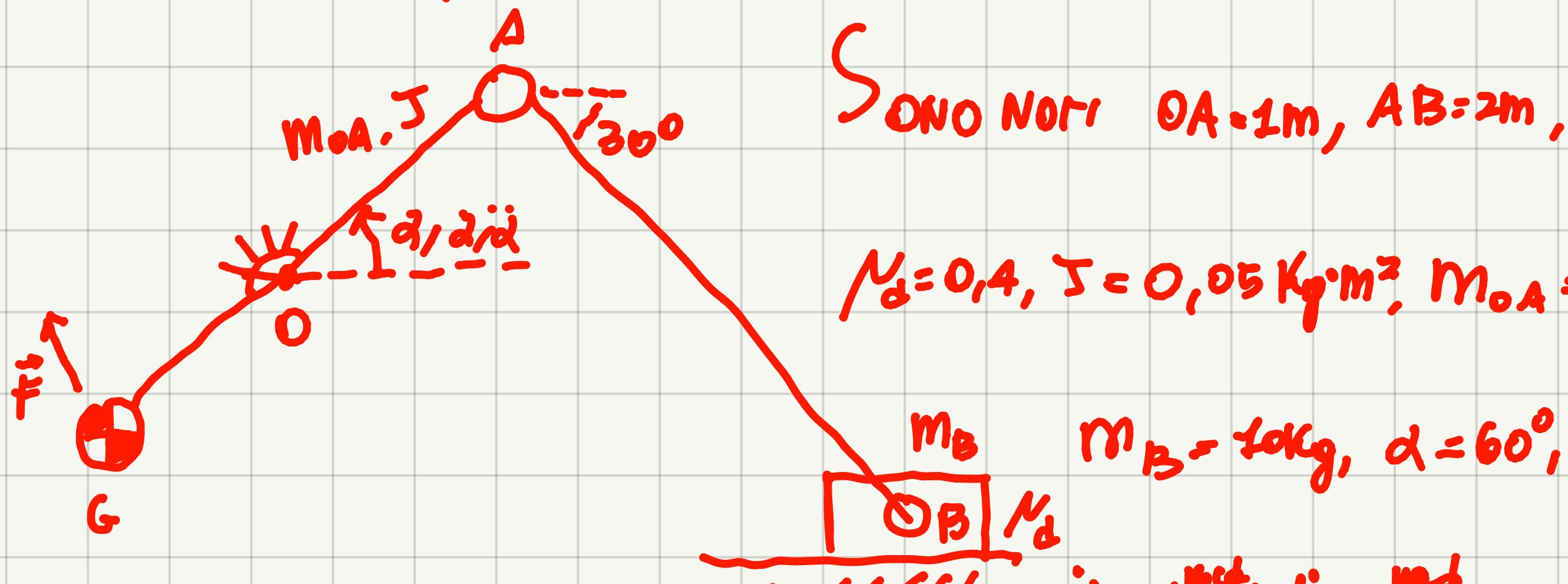
$$-m a_B - H_b - F = 0$$

$$N - V_B = 0$$

$$M = 0$$

SI CONSIDERI IL SEGUENTE SISTEMA COMPOSTO DA:

- ASTA OA DORATA DI MASSA  $M_{OA}$  E CON BARICENTRO IN G
- ASTA AB DI MASSA TRASCURABILE
- BLOCCO IN B DI MASSA  $M_B$  CHE SCORRE SU UN PIANO E SOGGETTA AD ATTRITO DINAMICO  $N_d$



SONO NOTI  $OA = 1m$ ,  $AB = 2m$ ,  $OG = 0,5m$

$$N_d = 0,4, \quad I = 0,05 \text{ kg} \cdot \text{m}^2, \quad M_{OA} = 2 \text{ kg}$$

$$M_B = 1 \text{ kg}, \quad d = 60^\circ,$$

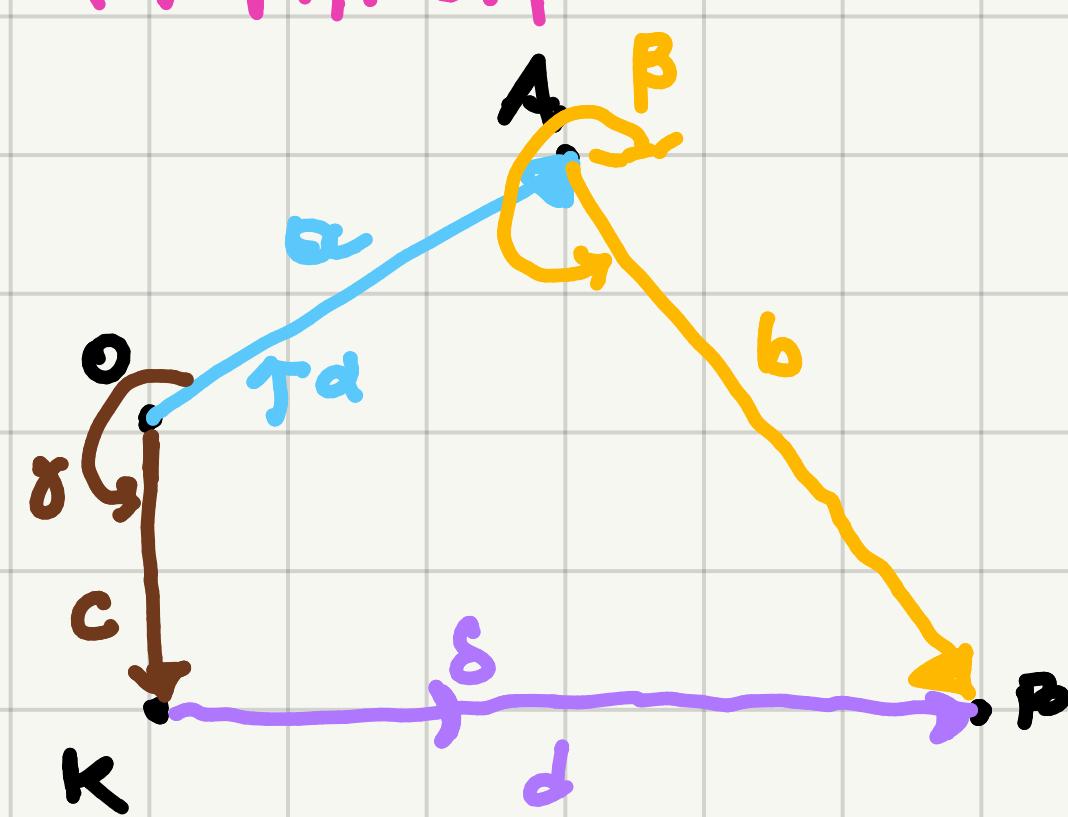
$$\dot{\alpha} = 1 \frac{\text{rad}}{\text{s}}, \quad \ddot{\alpha} = 1 \frac{\text{rad}}{\text{s}^2}$$

SI CALCOLINO:  $\vec{v}_B, \vec{\alpha}_B, \vec{F}$

ESSENDO IN MOTTO, CI ASPETTIAMO  $n \geq 1$

$$n = (3 \cdot 3) - (2_0 + 2_A + 2_B + 2_{\text{partito}}) = 1 \quad \checkmark$$

CINEMATICA



$$\vec{a} + \vec{b} = \vec{c} + \vec{d}$$

- a NOTO  $\alpha, \dot{\alpha}, \ddot{\alpha}$  NOTO
- b NOTO  $\beta = 360 - 30 = 330^\circ$   
 $\dot{\beta} \neq 0; \ddot{\beta} \neq 0$
- c?  $\gamma = 270^\circ$  FISSO
- d?  $\delta = 0^\circ$  FISSO

$$\left\{ a \cos \alpha + b \cos \beta = d \right.$$

$\Rightarrow$

$$\left. a \sin \alpha + b \sin \beta = -c \right.$$

$$\left\{ d = 2,23 \text{ m} \right.$$

$$\left. c = 0,434 \text{ m} \right.$$

$$\left\{ \begin{array}{l} -a \dot{\alpha} \sin \alpha - b \dot{\beta} \sin \beta = d \\ a \dot{\alpha} \cos \alpha + b \dot{\beta} \cos \beta = 0 \end{array} \right.$$

$$\rightarrow \left\{ \begin{array}{l} \dot{\alpha} = -1,155 \text{ rad/s} \\ \dot{\beta} = -0,289 \text{ rad/s} \end{array} \right. \Rightarrow \vec{v}_B = -1,155 \text{ m/s}$$

ANTICAPIAMO, PER LA PARTE DI DINAMICA, IL CALCOLO DI  $\vec{V}_G$

$$\vec{V}_G = \vec{V}_0 + \dot{d} \vec{OG} \hat{E} = 0,5 \hat{E} \text{ m/s}$$

$$= 0,5 \text{ m/s} ( -\cos \theta \hat{i} + \sin \theta \hat{j} ) = (0,433 \hat{i} - 0,25 \hat{j}) \text{ m/s}$$

$$\begin{cases} -\alpha d \sin \theta - \alpha d^2 \cos \theta - b \ddot{p} \sin \beta - b \dot{p}^2 \cos \beta = \ddot{d} \\ \alpha \ddot{d} \cos \theta - \alpha d^2 \sin \theta + b \ddot{p} \cos \beta - b \dot{p}^2 \sin \beta = 0 \end{cases}$$

$$\begin{cases} \ddot{d} = -1,3476 \text{ m/s}^2 \\ \ddot{p} = 0,1631 \text{ rad/s} \end{cases} \Rightarrow \vec{\alpha}_P = -1,3476 \hat{z} \text{ m/s}^2$$

$$\vec{\alpha}_G = \ddot{d} \vec{OG} \hat{E} + \dot{d}^2 \vec{OG} \hat{n} = (0,5 \hat{i} + 0,5 \hat{j}) \text{ m/s}^2$$

## DINAMICA

$$\frac{d}{dt} K = \frac{d}{dt} K^{(OA)} + \frac{d}{dt} K^{(B)}$$

- $\frac{d}{dt} K^{(OA)} = M_{OA} (\vec{\alpha}_G \cdot \vec{v}_G) + J (\vec{\omega} \cdot \vec{\dot{\omega}})$   
 $= M_{OA} \alpha_G v_{Gx} + M_{OA} \cancel{\alpha_B v_{Bx}} + J \ddot{d}$

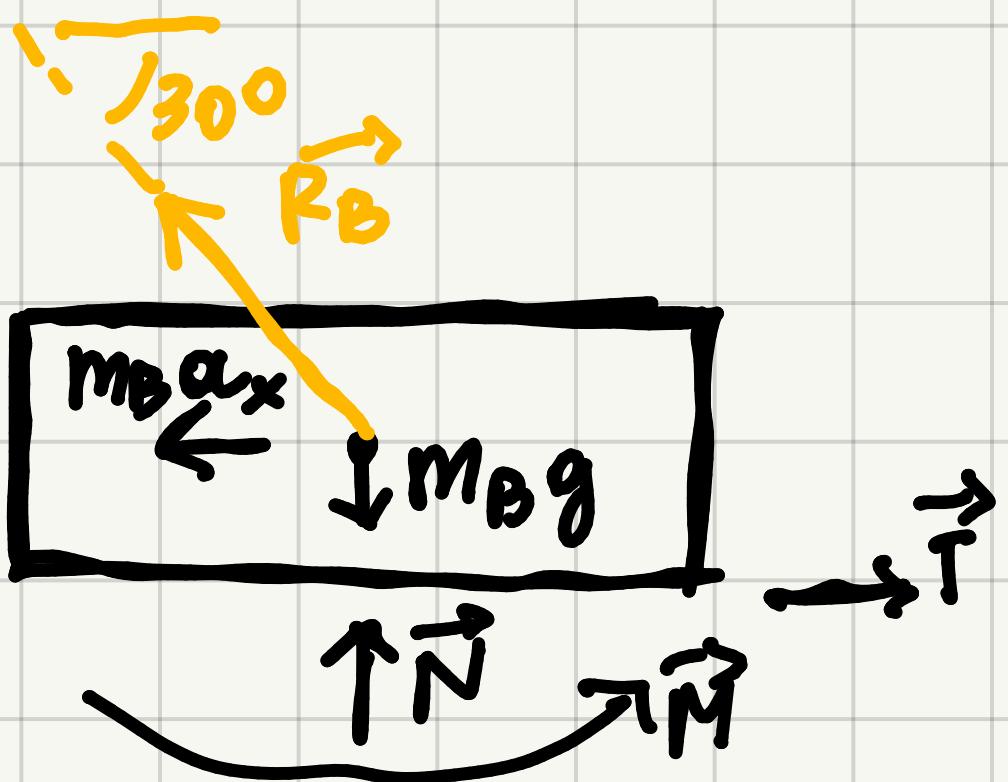
- $\frac{d}{dt} K^{(B)} = M_B (\vec{\alpha}_B \cdot \vec{v}_B) = M_B \alpha_B v_B$

NON SI CONSIDERA  
L'UNICO VERTICE

$$\sum P_{\text{ATTIVE}} = P^{(\text{LAG})} + \cancel{P^{(\text{PBS})}} + P^{(\text{F})} + P^{(\text{ARREDO})} =$$

$$= -M_{OA} g v_{Gy} - F_{Ax} - T |v_B|$$

$$\vec{T} = -N_d N \frac{\vec{v}_{\text{ref}}}{|\vec{v}_{\text{ref}}|} \Rightarrow |\vec{T}| = -N_d N$$



ASTA SCARICA  $\Rightarrow R_B$  INCUNATA DI  $30^\circ$   
 L'ASTA AB HA  $M \approx 0$

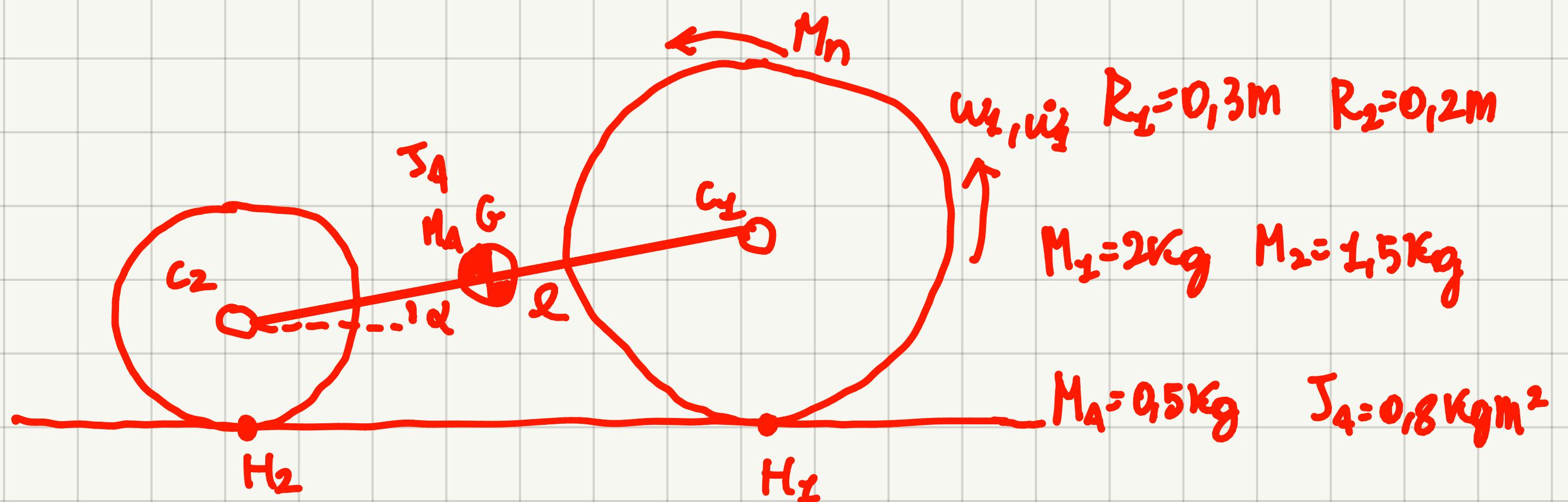
$$\begin{cases} T - m_B a_{Bx} - R_B \cos 30 = 0 \\ N - m_B g + R_B \sin 30 = 0 \\ T = N_d N \end{cases} \Rightarrow \begin{cases} N = 73,374 \text{ N} \\ T = 29,354 \text{ N} \\ R_B = \text{QUALCOSA DI IRRILEVANTE} \end{cases}$$

$$\sum P_{\text{ATTIVE}} = \frac{d}{dt} K \Rightarrow F = -90,21 \text{ N}$$

$$\Rightarrow M_{01} \alpha_{01} V_{0x} + J \ddot{\alpha} \dot{x} + m_B a_B V_B = -M_{01} g V_{0y} - F V_{0y} - T V_B$$

SI CONSIDERI IL SISTEMA RAFFIGURANTE UN TRATTORE IN MOVIMENTO,

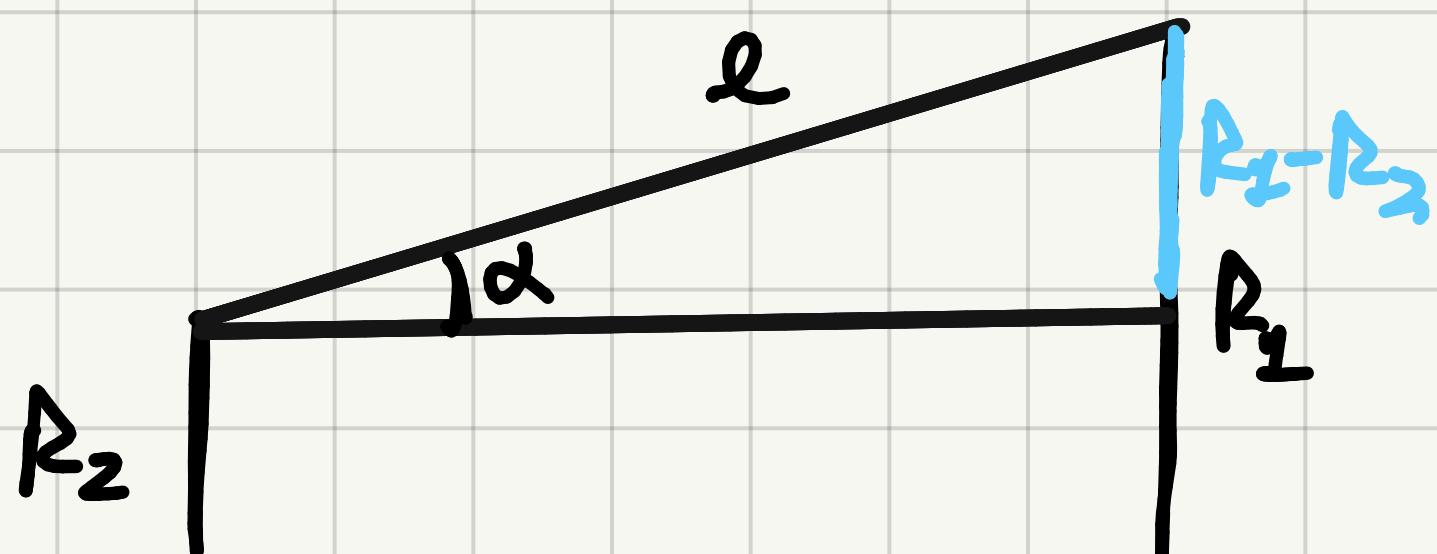
OSSERVANDO IN PARTICOLARE LE DIVERSE GRANDEZZE DELLE RUOTE



$$J_1 = 0.5 \text{ kg m}^2 \quad J_2 = 0.8 \text{ kg m}^2 \quad \omega_1 = 1.5 \text{ rad/s} \quad \omega_2 = 0.5 \text{ rad/s} \quad d = 5^\circ \quad N_1 = 0.8 \quad N_2 = 0.5 \quad N_3 = 0.03$$

CALCOLARE  $M_n$  E  $H_{H_2}$  IN CASI NELLE INDICATE

$$n = 3 \cdot 3 - (2c_2 + 2c_1 + 2_{H_2} + 2_{H_1}) = 1$$



$$R_1 - R_2 = l \sin \alpha \Rightarrow l = \frac{R_2 - R_1}{\sin \alpha}$$

$$\vec{V}_2 = \vec{V}_{H_2} + \vec{\omega}_2 \times (\vec{c}_2 - \vec{h}_2) = \vec{\omega}_2 \hat{R} \cdot \vec{R}_2 \hat{x} = -\vec{\omega}_2 R_2 \hat{x}$$

! NON POSSO APPLICARE RIVIUS SUI PUNTI DI CONTATTO,  
MA VALE SEMPRE CHE  $\vec{\alpha} = \frac{d}{dt} \vec{\omega}$

$$\vec{\alpha}_2 = \frac{d}{dt} \vec{V}_2 = \frac{d}{dt} (-\vec{\omega}_2 R_2 \hat{x}) = -\dot{\vec{\omega}}_2 R_2 \hat{x}$$

$\vec{V}_2$   $\leftarrow$  CIR  $\rightarrow +\infty \Rightarrow$  ASTA IN MOTORE ROTAZIONALE

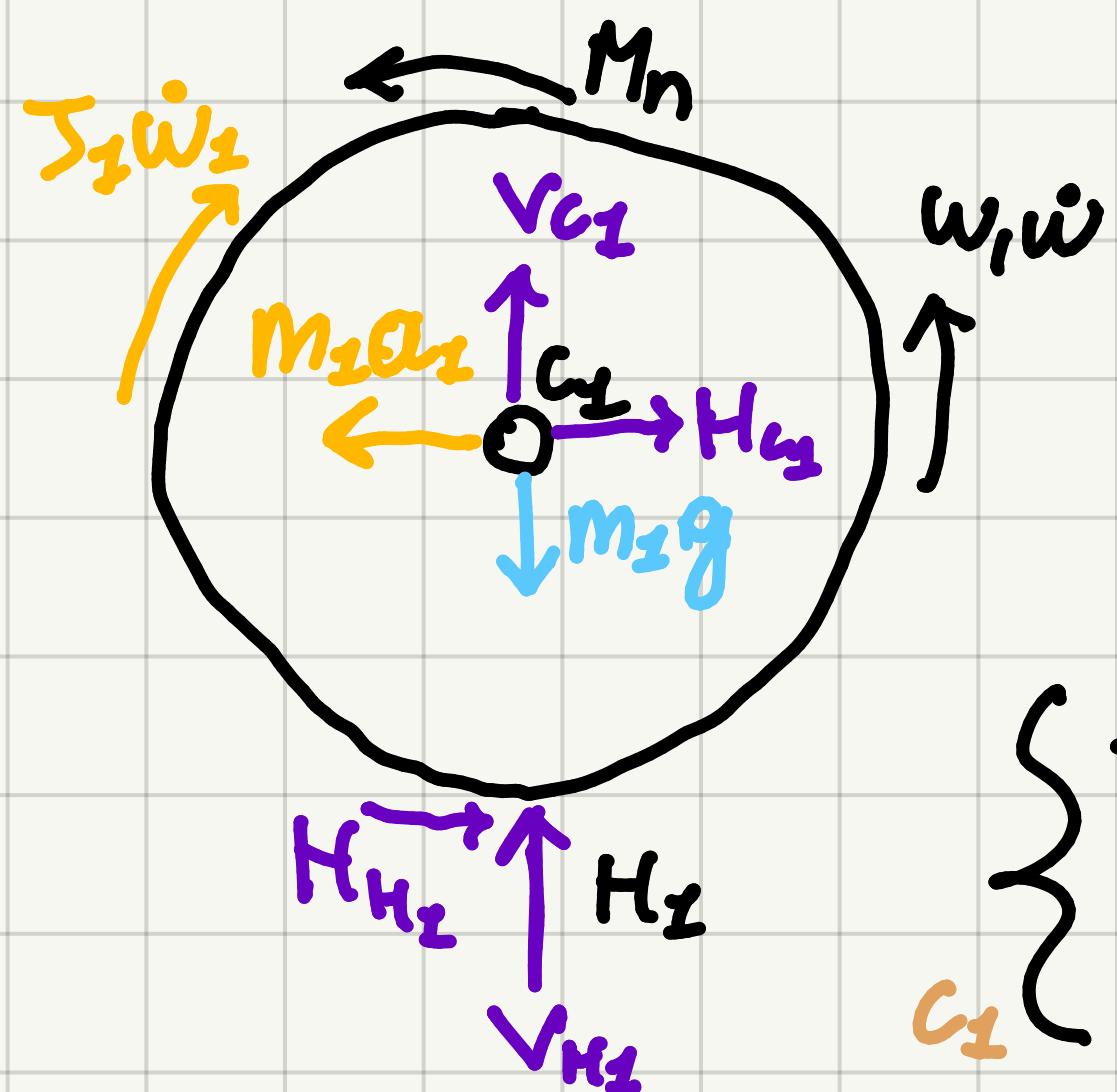
$$\Rightarrow \omega_A = 0, \dot{\omega}_A = 0 \Rightarrow \vec{V}_G = \vec{V}_2, \vec{\alpha}_G = \vec{\alpha}_2$$

$$\vec{V}_2 \quad c_2 \quad \uparrow \alpha \quad \text{MA ANCHE } \vec{V}_2 = \vec{V}_1 = -\vec{\omega}_1 R_1 \hat{x}$$

INOLTRE,  $\vec{V}_2 = \vec{V}_{H_2} + \vec{\omega}_2 \hat{K} \times (\vec{c}_2 - \vec{h}_2) \Rightarrow \omega_2 = \frac{R_2}{R_1} \omega_1 \rightarrow \dot{\omega}_2 = \frac{R_2}{R_1} \dot{\omega}_1$

INTUITIVAMENTE, A PARITÀ DI  $\vec{V}$ , SE  $R$  AUMENTA  $\omega$  DIMINUISCE

CASO I:  $N_{Vr}=0$



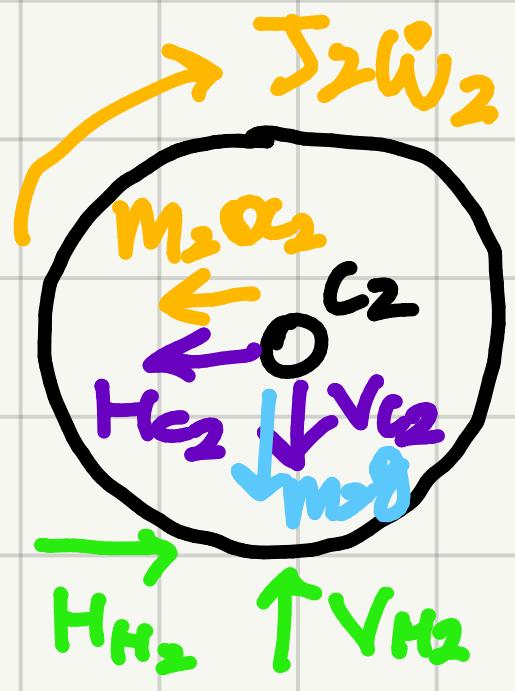
DISEGNO FORZE E COPIES DI INERZIA  
OPPOSTE ALLA CONVENZIONE

$$\begin{cases} -m_1 \alpha_1 + H_{C1} + H_{H1} = 0 \\ V_{C1} + V_{H1} - m_1 g = 0 \\ M_n - J_1 \dot{\omega}_1 + H_{H1} R_1 = 0 \end{cases}$$



! 5 INCognITE... SERVONO ALTRÉ EQUAZIONI  $\Rightarrow$  USO DI ALTRI CORPI

$$\begin{cases} -H_{C1} - H_{C2} - M_A \alpha_2 = 0 \\ -V_{C1} + V_{C2} - M_A g = 0 \\ M_A g l_2 \cos \alpha - m_1 a_2 \frac{l}{2} \sin \alpha - V_{C2} l \cos \alpha \\ + H_{C2} l \sin \alpha = 0 \end{cases}$$

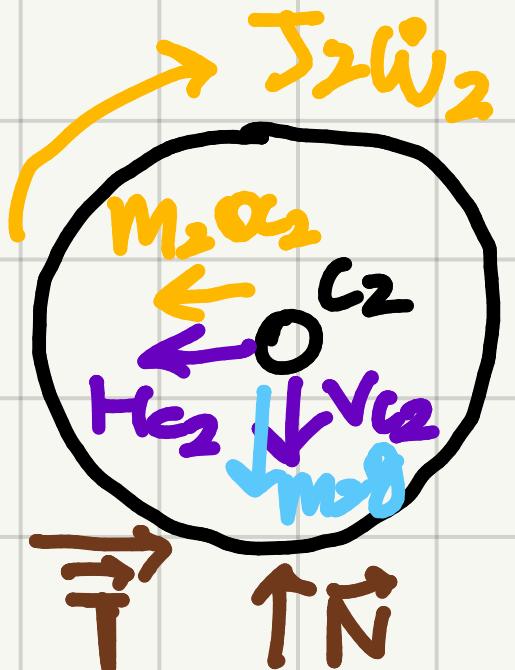


$$\left\{ \begin{array}{l} -H_{C2} - M_2 \alpha_2 + H_{M2} = 0 \\ -V_{C2} - M_2 g + V_{H2} = 0 \\ -J_2 \ddot{\omega}_2 + H_{M2} R_2 = 0 \end{array} \right.$$

9 EQUAZIONI  
9 INCognITE ✓

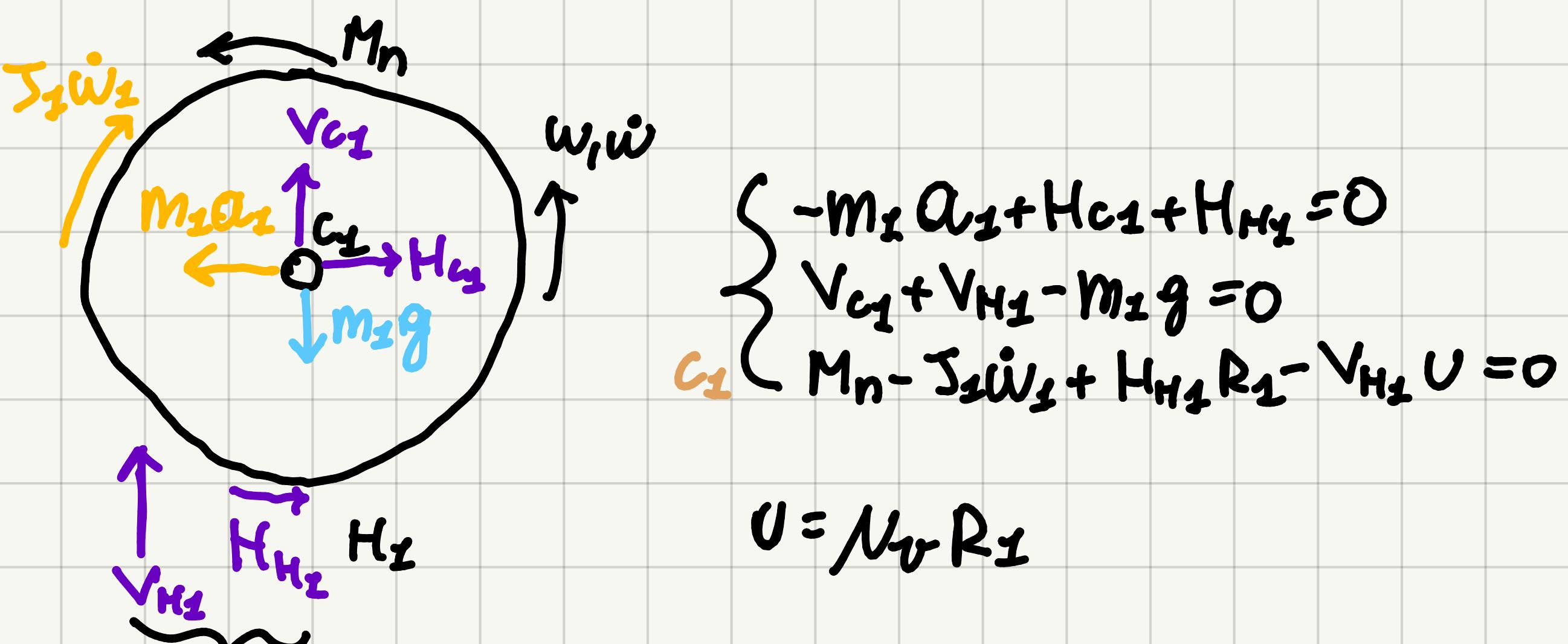
$$\Rightarrow H_{M2} = \frac{J_2 \ddot{\omega}_2}{R_2} = 1,125 \text{ N} \quad M_n = 0,768 \text{ N}\cdot\text{m}$$

VERIFICA DI ADEGUENZA:  $T \leq T_{elim}$ ?



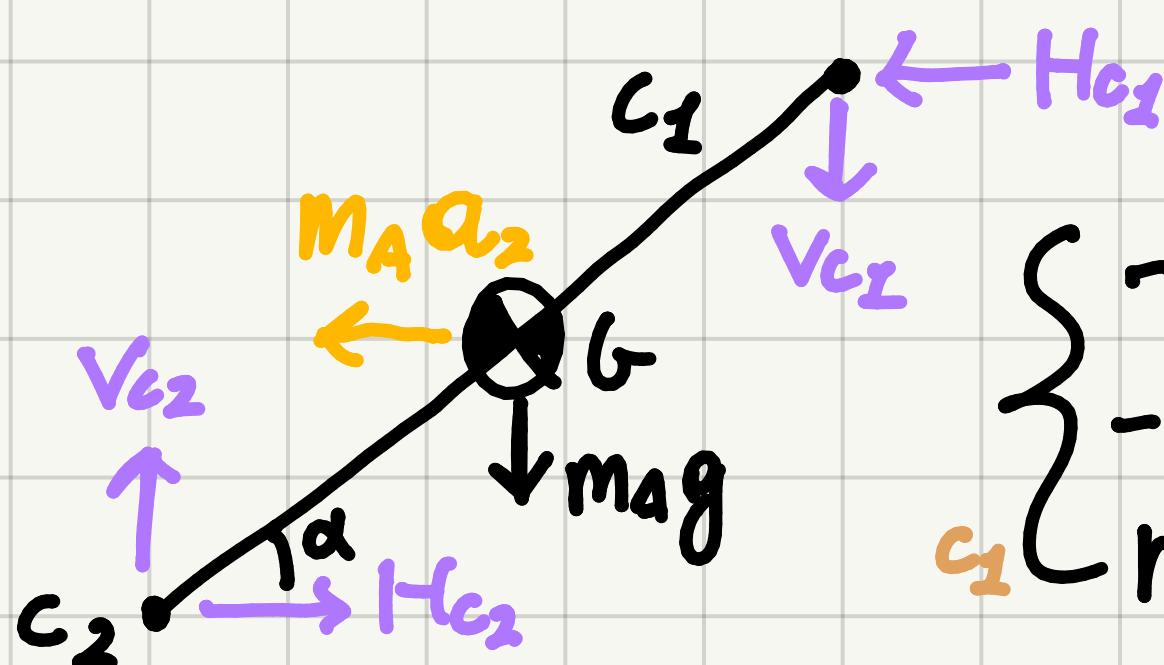
$$\begin{aligned} & \text{SI NOTI CHE } T = H_{M2} \text{ E } N = V_{H2} \\ & \Rightarrow \begin{cases} T = 1,125 \text{ N} \\ N = 17,29 \text{ N} \end{cases} \\ & T_{elim} = N_s \cdot N = 13,38 \Rightarrow T \leq T_{elim} \checkmark \end{aligned}$$

CASO 1:  $N_s = 0,03$



$$U = N_s \cdot R_1$$

$V$   $V_{H2}$  SPOSTATO A SINISTRA PER GENERARE UN MOTORE IN SENSO OPARIO  
E QUINDI FADE RESISTENZA



$$\left\{ \begin{array}{l} -H_{C1} - H_{C2} - M_A \alpha_2 = 0 \\ -V_{C1} + V_{C2} - M_A g = 0 \\ M_A g l_2 \cos \alpha - m_1 a_2 \frac{\pi}{2} \sin \alpha - V_{C2} l \cos \alpha \\ + H_{C2} l \sin \alpha = 0 \end{array} \right.$$



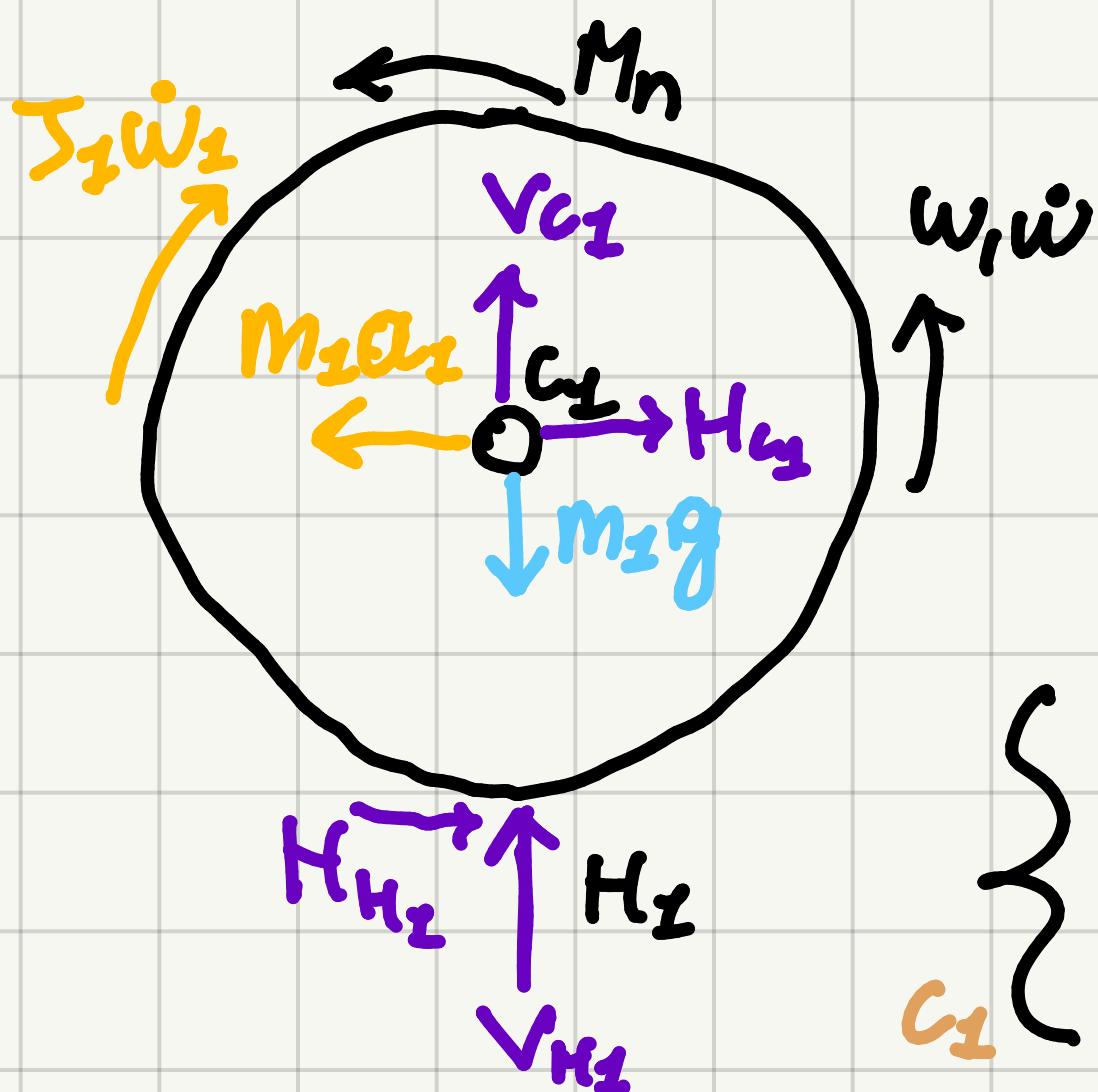
$$\left\{ \begin{array}{l} -H_{c2} - M_2 \alpha_2 + H_{H2} = 0 \\ -V_{c2} - M_2 g + V_{H2} = 0 \\ -J_2 \dot{\omega}_2 + H_{H2} R_2 - V_{H2} U = 0 \end{array} \right.$$

$$U = N_d R_2 \Rightarrow M_n = 1,12 \text{ Nm} \quad H_{H2} = 1,64 \text{ N}$$

SI NOM L'ASPERTAIO AUMENTO DI  $M_n$  DOVUTO AI NUOVI CONTRIBUTI

CASO 3:  $N_d = 0, N_d = 0,5$  SOLO SU  $D_2$

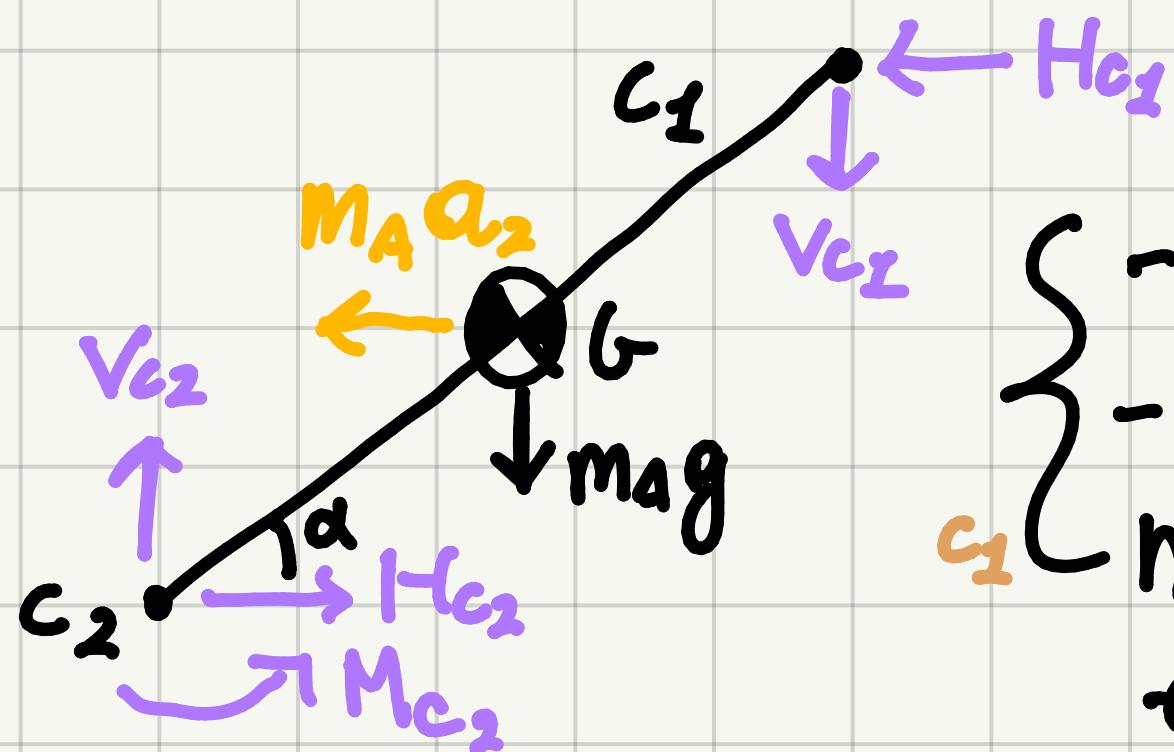
IMPROVVISAMENTE,  $D_2$  SI BLOCCA



DISEGNO FORZE E COPIES DI INERZIA

OPPOSTE ALLA CONVENZIONE

$$\left\{ \begin{array}{l} -M_1 \alpha_1 + H_{c1} + H_{H1} = 0 \\ V_{c1} + V_{H1} - M_1 g = 0 \\ C_1 - M_n - J_1 \dot{\omega}_1 + H_{H1} R_1 = 0 \end{array} \right.$$



$$\left\{ \begin{array}{l} -H_{c1} - H_{c2} - M_A \alpha_2 = 0 \\ -V_{c2} + V_{c1} - M_A g = 0 \\ C_1 - M_A g l_2 \cos \alpha - m_2 \alpha_2 \frac{l}{2} \sin \alpha - V_{c2} l \cos \alpha \\ + H_{c2} l \sin \alpha + M_c = 0 \end{array} \right.$$

ORA ABBIANO 9 EQUAZIONI E 10 INCognITE  $\Rightarrow$  LA 10MA EQUAZIONE  
È DATA DAL MODELLO DI ATTRAIO DINAMICO



$$\left\{ \begin{array}{l} H_{H2} - H_{c2} - M_2 \alpha_2 = 0 \\ -V_{c2} - M_2 g + V_{H2} = 0 \\ -M_{C2} + H_{H2} R_2 = 0 \end{array} \right.$$

! SI NOM CHE, NON  
ESSENDO PIÙ IN  
ROTAZIONE,  $C_{in} = 0$

$$H_{H2} = N_d \cdot |V_{H2}|$$

## RIPESSAMO I TRE ESEMPI COL BILANCIAMENTO DI POTENZE

### CASO 1

$$\frac{d}{dt} K = \frac{d}{dt} K^{(D_1)} + \frac{d}{dt} K^{(A_{S1})} + \frac{d}{dt} K^{(D_2)}$$

- $\frac{d}{dt} K^{(D_1)} = M_1 \vec{\alpha}_1 \cdot \vec{v}_1 + J_1 \vec{\omega} \cdot \vec{\omega} = M_1 (-R_2 \dot{\omega}_2 \hat{i}) \cdot (-R_2 \omega_2 \hat{x}) + J_1 (\dot{\omega}_2 \hat{k}) (\omega_2 \hat{r})$ .
- $(\omega_2 \hat{r}) = M_1 R_2^2 \dot{\omega}_2 \omega_2 + J_1 \dot{\omega}_2 \omega_2$
- $\frac{d}{dt} K^{(A_{S1})} = M_1 (\vec{\alpha}_0 \cdot \vec{v}_0) = M_1 R_2^2 \dot{\omega}_2 \omega_2$
- $\frac{d}{dt} K^{(D_2)} = M_2 \vec{\alpha}_2 \cdot \vec{v}_2 + J_2 \vec{\omega}_2 \cdot \vec{\omega}_2 = M_2 (-R_2 \dot{\omega}_2 \hat{i}) \cdot (G R_2 \omega_2 \hat{x}) + J_2 (\dot{\omega}_2 \hat{k}) (\omega_2 \hat{r}) = M_2 R_2^2 \dot{\omega}_2 \omega_2 + J_2 \dot{\omega}_2 \omega_2$

MOTORE FORZE PESO

$$\sum P_{ATTIVE} = \cancel{P_{PESO}^{(D_1)}} + \cancel{P_{PESO}^{(A_{S1})}} + \cancel{P_{PESO}^{(D_2)}} + P_{MOT} = M_n \cdot \omega_2$$

$$\frac{d}{dt} K = \sum P_{ATTIVE} \Rightarrow M_n = 0,768 \text{ N} \cdot \text{m}$$

PER RISOLVERE  $H_{H_2}$  OCCORRE PER FORZA USARE L'EQUILIBRIO DINAMICO

### CASO 2

PER L'ENERGIA CINETICA ABBIANO LE STESE EQUAZIONI DEL CASO 1

$$\frac{d}{dt} K = \frac{d}{dt} K^{(D_1)} + \frac{d}{dt} K^{(A_{S1})} + \frac{d}{dt} K^{(D_2)}$$

- $\frac{d}{dt} K^{(D_2)} = M_1 \vec{\alpha}_1 \cdot \vec{v}_1 + J_1 \vec{\omega} \cdot \vec{\omega} = M_1 (-R_2 \dot{\omega}_2 \hat{i}) \cdot (-R_2 \omega_2 \hat{x}) + J_1 (\dot{\omega}_2 \hat{k}) \cdot (\omega_2 \hat{k})$
- $(\omega_2 \hat{k}) = M_1 R_2^2 \dot{\omega}_2 \omega_2 + J_1 \dot{\omega}_2 \omega_2$
- $\frac{d}{dt} K^{(CASI)} = M_a (\vec{\alpha}_0 \cdot \vec{v}_0) = M_a R_2^2 \dot{\omega}_2 \omega_2$
- $\frac{d}{dt} K^{(D_2)} = M_2 \vec{\alpha}_2 \cdot \vec{v}_2 + J_2 \vec{\omega} \cdot \vec{\omega} = M_2 (-R_2 \dot{\omega}_2 \hat{i}) \cdot (G R_2 \omega_2 \hat{x}) + J_2 (\dot{\omega}_2 \hat{k}) (\omega_2 \hat{k}) = M_2 R_2^2 \dot{\omega}_2 \omega_2 + J_2 \dot{\omega}_2 \omega_2$

MOTO E FORZE PESO

$$\sum P_{ATTIVE} = \cancel{P_{PESO}^{(D_2)}} + \cancel{P_{PESO}^{(CASI)}} + \cancel{P_{PESO}^{(D_2)}} + P_{MOTO} + P_{RESISTENZA}^{(D_2)} + P_{RESISTENZA}^{(D_2)}$$

$$= M_n W_1 - N_a R_2 |V_{H_2} \omega_2| - N_a R_2 |V_{H_2} \omega_2|$$

ESSENDO L'ATTRITO DISSIPATIVO, PONGO IL SEGNO "-" DAVANTI. PER

EVITARE DI FAR CASINO COI SEGNI, USO I MODULI

$$\frac{d}{dt} K = \sum P_{ATTIVE} \quad \text{MA HO 3 INCognITE} \Rightarrow \text{SERVE L'EQUILIBRIO DINAMICO}$$

CASO 3

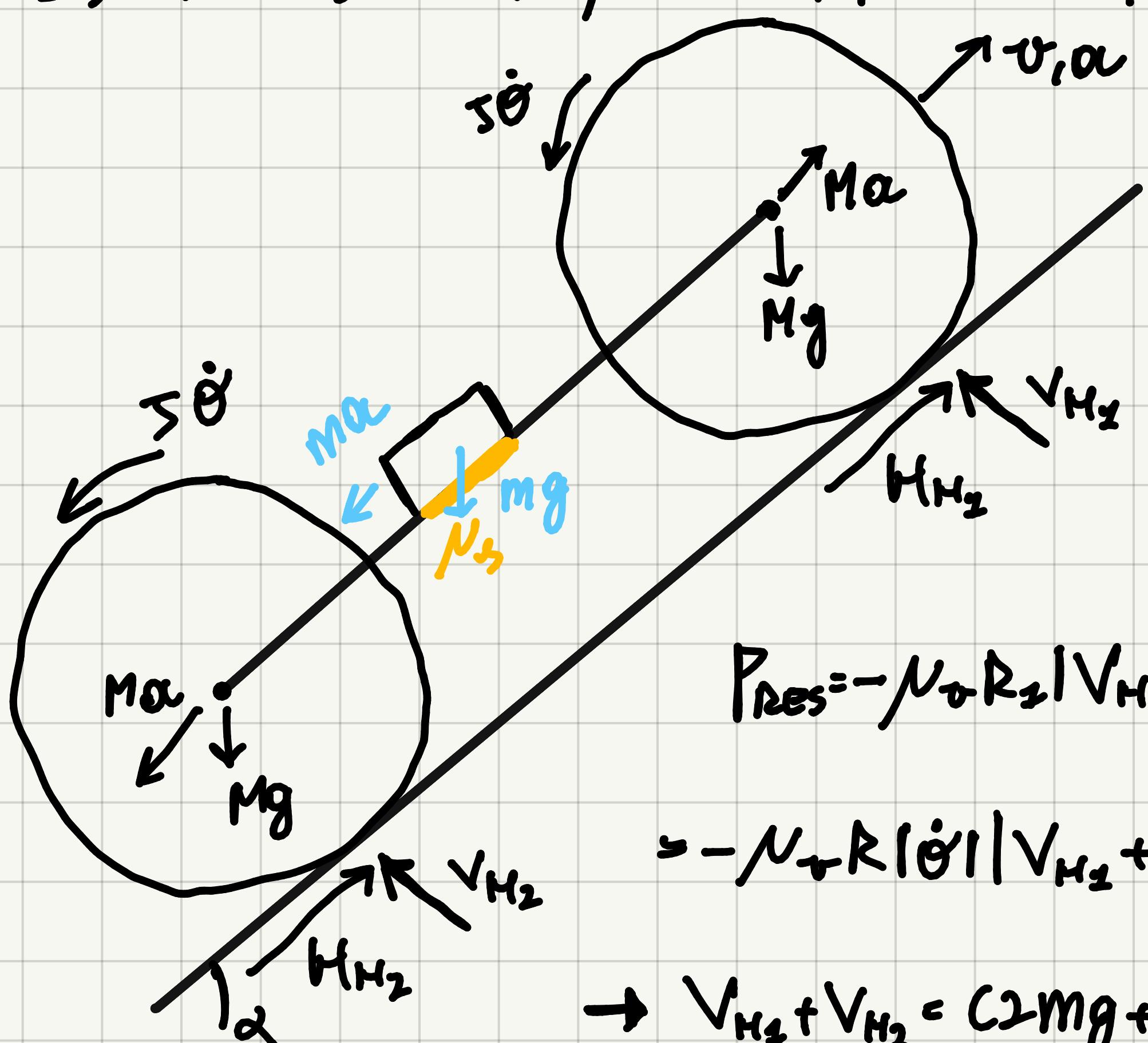
- $\frac{d}{dt} K^{(D_2)} = M_1 \vec{\alpha}_1 \cdot \vec{v}_1 + J_1 \vec{\omega} \cdot \vec{\omega} = M_1 (-R_2 \dot{\omega}_2 \hat{i}) \cdot (-R_2 \omega_2 \hat{x}) + J_1 (\dot{\omega}_2 \hat{k}) \cdot (\omega_2 \hat{k})$
- $(\omega_2 \hat{k}) = M_1 R_2^2 \dot{\omega}_2 \omega_2 + J_1 \dot{\omega}_2 \omega_2$
- $\frac{d}{dt} K^{(CASI)} = M_a (\vec{\alpha}_0 \cdot \vec{v}_0) = M_a R_2^2 \dot{\omega}_2 \omega_2$
- $\frac{d}{dt} K^{(D_2)} = M_2 \vec{\alpha}_2 \cdot \vec{v}_2 = M_2 R_2^2 \dot{\omega}_2 \omega_2$

MOTO E FORZE PESO

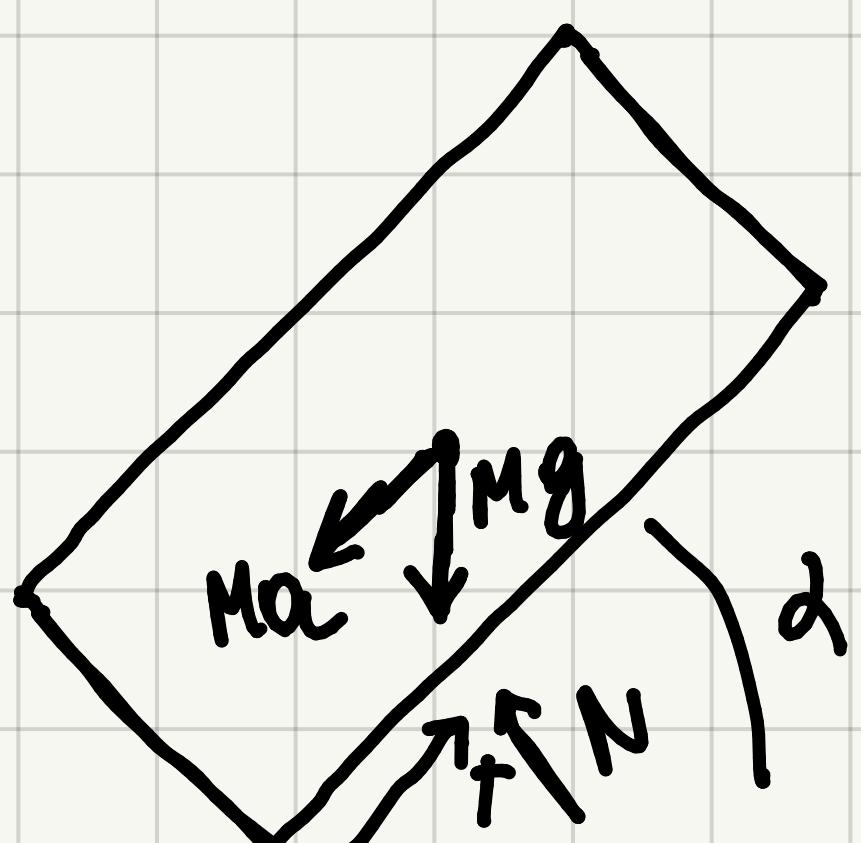
$$\sum P_{ATTIVE} = \cancel{P_{PESO}^{(D_2)}} + \cancel{P_{PESO}^{(CASI)}} + \cancel{P_{PESO}^{(D_2)}} + P_{MOTO} + P_{RESISTENZA}^{(D_2)} = M_n W_1 - N_a R_2 |V_{H_2} \omega_2|$$

## INFO EXTRA SU CONDIZIONI DI ADERENZA

### 1) RUOTE UGUALI; MOTO DI PURO ROTOLAMENTO



### 2) CALCOLO $a_{elim}$



$$\begin{cases} T - Ma - Mg \sin \alpha = 0 \\ -Mg \cos \alpha + N = 0 \\ T_{elim} = N_s N \end{cases}$$

IN CONDIZIONI LIMITE,  $T = T_{elim}$

$$\Rightarrow T_{elim} - Ma_{elim} - Mg \sin \alpha = 0 \Rightarrow a_{elim} = \frac{T_{elim} - Mg \sin \alpha}{M}$$