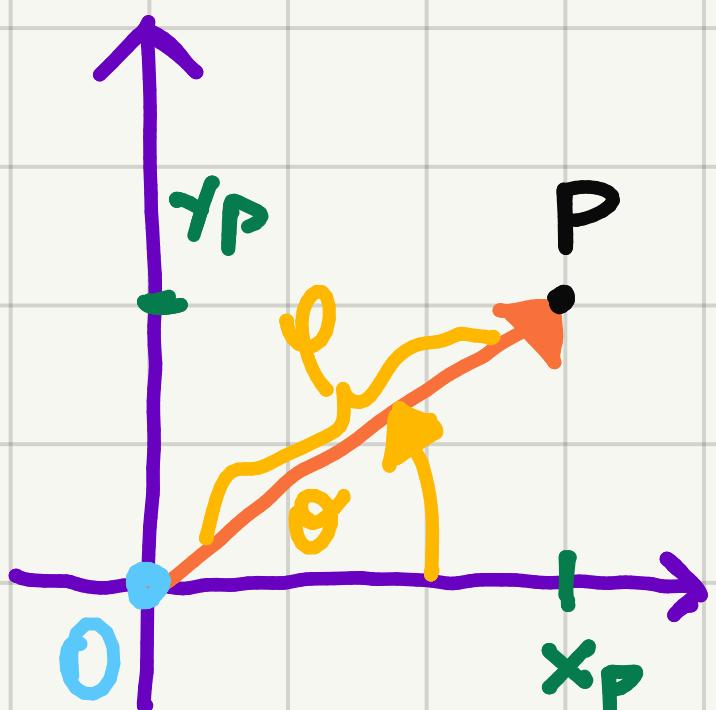


CINEMATICA PUNTO MATERIALE

DEFINIZIONE \vec{P} IN FUNZIONE DI UN SISTEMA DI RIFERIMENTO (= OSSERVATORE)

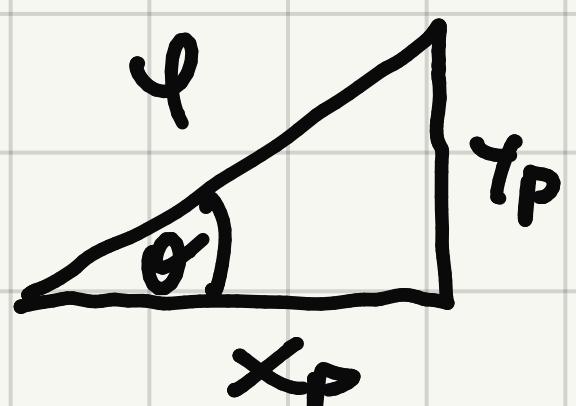


E UNA SUA ORIGINE

COORDINATE CARTESIANE $\vec{P} = x_P \hat{x} + y_P \hat{y}$

COORDINATE POLARI $\vec{P} = \rho e^{i\theta}$ $\theta = \text{ANOMALIA}$

RELAZIONI CARTESIANE/POLARI



$$\rho = \sqrt{x_P^2 + y_P^2}$$

$$y_P = x_P \tan \theta \Rightarrow \theta = \arctan \frac{y_P}{x_P}$$

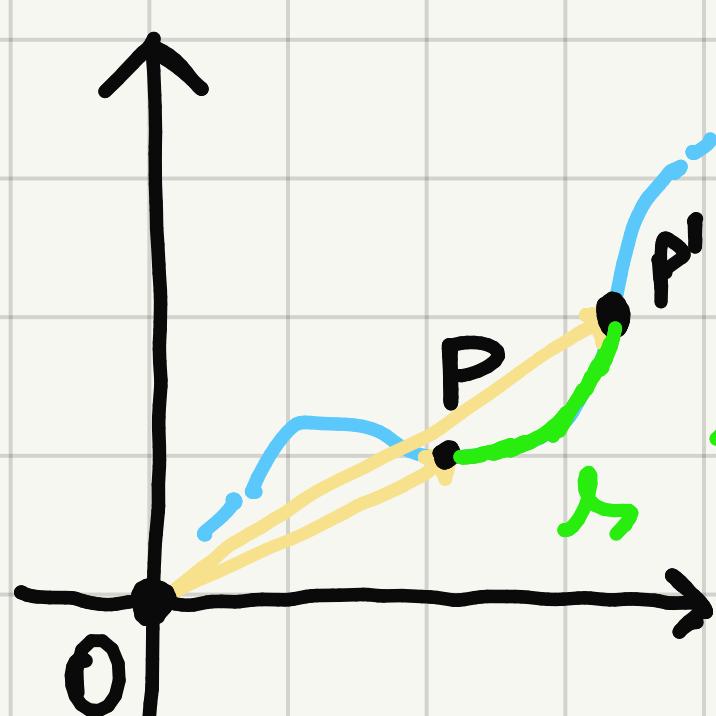
$$x_P = \rho \cos \theta \quad y_P = \rho \sin \theta \quad \text{RELAZIONE DI EULER} \quad e^{i\theta} = \cos \theta + i \sin \theta$$

$$\text{ANGOLI PARTICOLARI} \quad e^{i0} = 1 \quad e^{i\pi/2} = i \quad e^{i\pi} = -1$$

SPOSTAMENTO

VARIAZIONE DELLE COORDINATE DI P ((x, y) , (ρ, θ)) IN FUNZIONE DI UN

PARAMETRO t. LUOGO GEOMETRICO DEI PUNTI OCCUPATI DA P → TRAIETTORIA



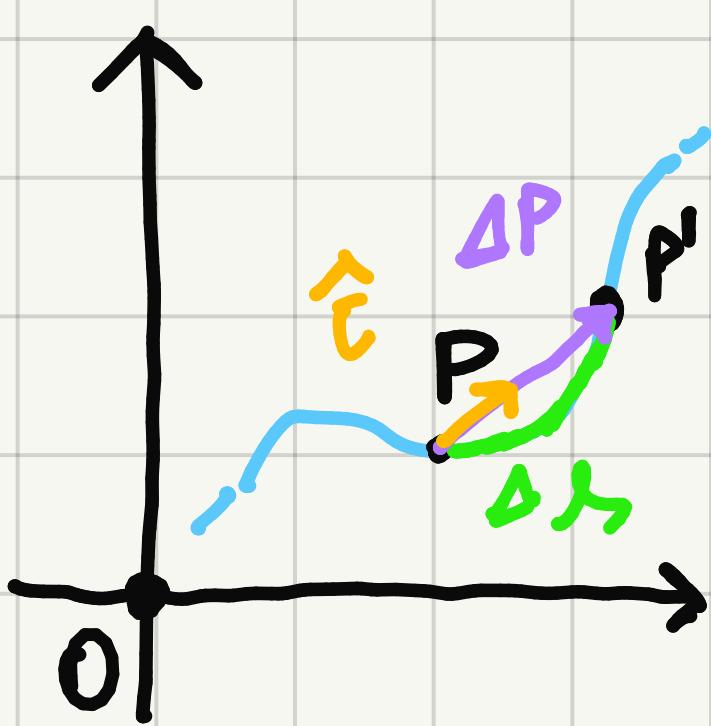
→ RELAZIONE TRA LE DUE COORDINATE

ASSUNSA CIRCONFERENZA $s \rightarrow$ DISTANZA, AL MOMENTO t, DI

P' DA P LUNGO LA TRAIETTORIA

$$P(t) \Rightarrow P(s(t))$$

VELOCITÀ



$$\vec{v} = \lim_{\Delta t \rightarrow 0} \frac{\Delta \vec{P}}{\Delta t} = \frac{d}{dt} \vec{P}(s(t)) = \frac{d\vec{P}}{ds} \cdot \frac{ds}{dt} = \dot{s} \hat{v}$$

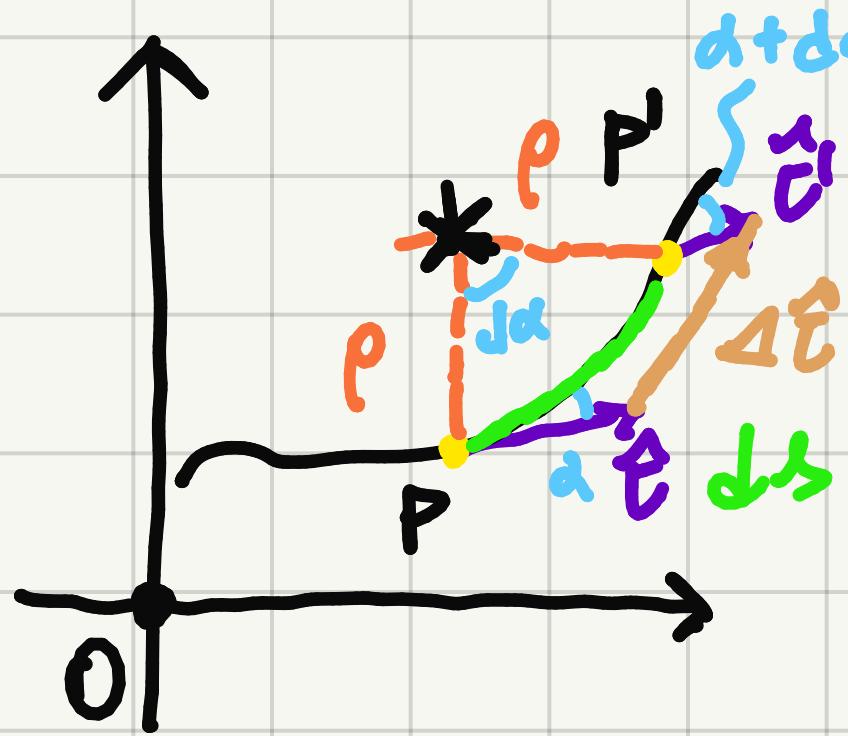
$$\frac{d\vec{P}}{ds} = \lim_{\Delta s \rightarrow 0} \frac{\vec{P}' - \vec{P}}{\Delta s} = \frac{\vec{P}(s + \Delta s) - \vec{P}(s)}{\Delta s}$$

$$\Delta t \rightarrow 0 \Rightarrow \Rightarrow \Delta P \approx \Delta s \Rightarrow \left| \frac{d\vec{P}}{ds} \right| = 1 \Rightarrow \frac{d\vec{P}}{ds} = \hat{v} \quad \vec{v} = \dot{s} \hat{v}$$

UN GENERICO VERSORE È IDENTIFICATO COME $\hat{v} = \frac{\vec{v}}{|\vec{v}|}$

ACCELERAZIONE

$$\vec{a} = \frac{d}{dt} \vec{v} = \frac{d}{dt} \left(\frac{d\vec{P}}{ds} \cdot \frac{ds}{dt} \right) = \frac{d^2 s}{dt^2} \cdot \frac{d\vec{P}}{ds} + \frac{ds}{dt} \cdot \frac{d}{dt} \frac{d\vec{P}}{ds} = \ddot{s} \hat{v} + \dot{s} \frac{d}{dt} \frac{d\vec{P}}{ds}$$

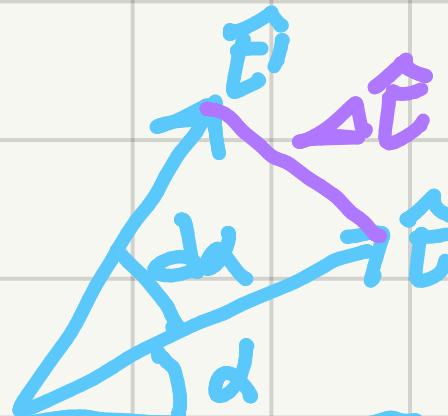


- SIANO P, P' PUNTI E TRAIETTORIA
- SI IDENTIFICHISI x
- SI IDENTIFICHISI α E LA VARIAZIONE Δs

$$\frac{d}{dt} \frac{d\vec{P}}{ds} = \frac{d}{dt} \cdot \frac{1}{\Delta s} \cdot \frac{d\vec{P}}{ds} = \mu \frac{d^2 \vec{P}}{ds^2} \Rightarrow \vec{a} = \ddot{s} \hat{v} + \dot{s}^2 \frac{d^2 \vec{P}}{ds^2}$$

$$\frac{d^2 \vec{P}}{ds^2} = \frac{d}{ds} \cdot \frac{d\vec{P}}{ds} = \frac{d}{ds} \hat{t} = \lim_{\Delta s \rightarrow 0} \frac{\Delta \hat{t}}{\Delta s}$$

- $d\vec{s} = \rho \Delta \alpha$, CON ρ RABBILO OSCULATORE



- $|\Delta \hat{t}| = |\hat{t}| \Delta \alpha$ PROIEZIONE $\Delta \hat{t} \perp \hat{t}$ PER $\Delta s \rightarrow 0$
 $\Rightarrow \Delta \hat{t} = \Delta \alpha \hat{n}$

$$\Rightarrow \frac{d}{ds} \hat{t} = \frac{d \alpha \hat{n}}{\rho \Delta \alpha} = \frac{\hat{n}}{\rho} \Rightarrow \vec{a} = \ddot{s} \hat{v} + \frac{\dot{s}^2}{\rho} \hat{n} = \vec{a}_{tan} + \vec{a}_{nor}$$

CASI PARTICOLARI:

• MOTO RETTILINEO $\Rightarrow \rho \rightarrow +\infty \Rightarrow \vec{a} = \vec{a}_{tan}$

• MOTO UNIFORME $\Rightarrow \ddot{s} = 0 \Rightarrow \vec{a} = \vec{a}_{nor}$

CINEMATICA CORPO RIGIDO

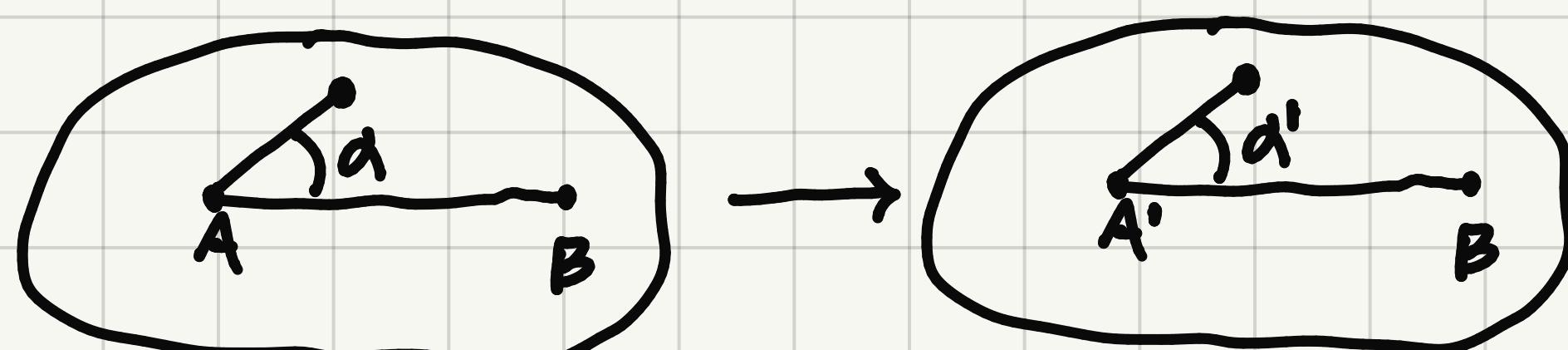
CORPO: INSIEME CONTINUO DI ∞ PUNTI DI FORMA E DIMENSIONE FINITI

STUDIO MOTO PIANO, CASO DI PUNTI I CUI s, v, α // PIANO DIRETTORE DEL

ANALISI AMO DI MOTO, SPOSTAMENTI INFINTESIMI PER CUI SI STUDIANO

$s(t), v(t), \alpha(t)$ A PUNTO NEL CORPO

CORPO RIGIDO: CORPO CHE NON SUBISCE DEFORMAZIONI. CONSEGUENZE:

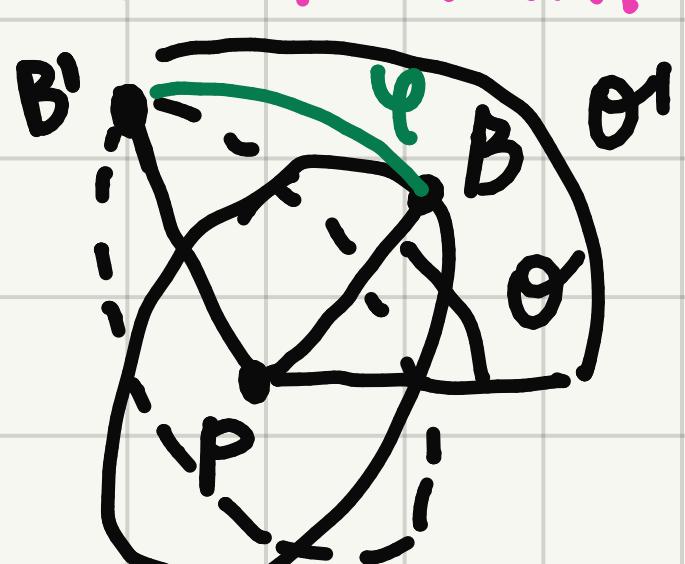


- $AB = A'B$ FISSO
- $d = d'$ FISSO

CORPO RIGIDO CARATTERIZZATO DA 3 GDL:

LE 2 COORDINATE SPAZIALI; ANOMALIA

MOTO TRASLATORIO $\rightarrow \theta$ COSTANTE. $P' = P(t + \Delta t) \forall P$ CORPO



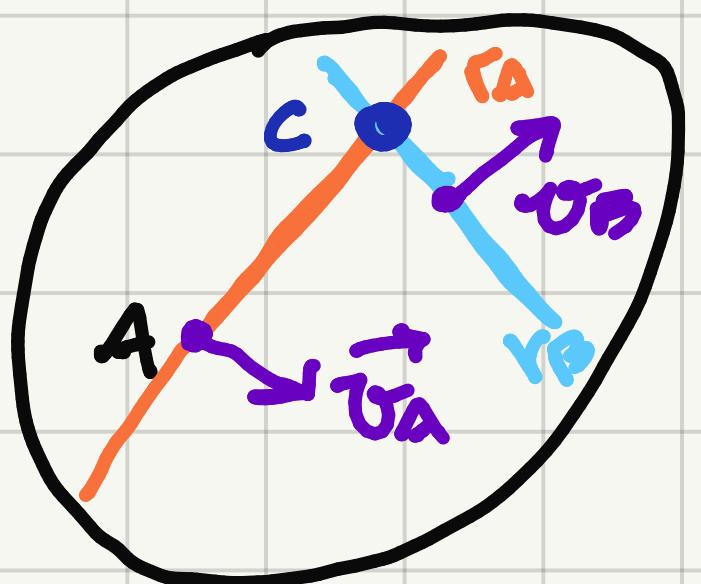
MOTO ROTATORIO $\rightarrow X_P, Y_P$ COSTANTI

P = CENTRO DI ROTAZIONE $\ell = \theta - \theta'$ $\vec{\varphi} = \vec{e}_R$

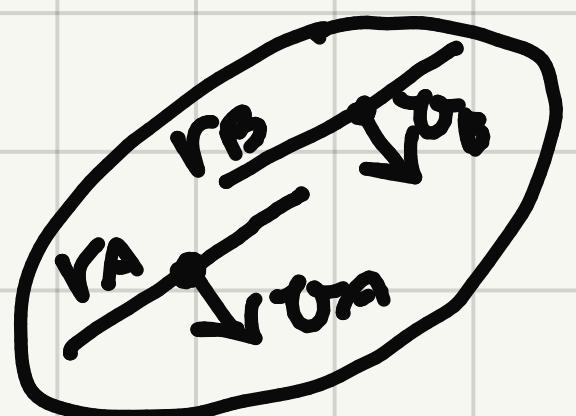
MOTO ROTOTRASLATORIO \rightarrow MOTO ROTATORIO + TRASLAZIONE DI P

NELL'ANALISI DELL'AMO DI MOTO, SI ESCUDE IL MOTO ROTOTRASLATORIO:

- \vec{v}_A, \vec{v}_B UGUALI IN MODULO, DIREZIONE, VERSO $\forall A, B$
- \exists CIR CI $\vec{v}_c = 0$

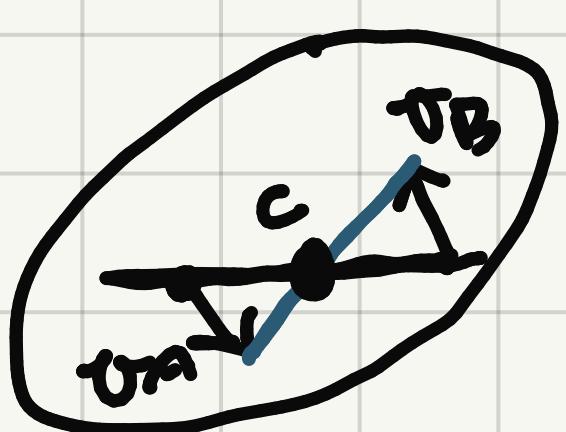


SIANO $\vec{v}_A \neq \vec{v}_B$
 SIANO $r_A \perp \vec{v}_A, r_B \perp \vec{v}_B$
 $\exists C | \vec{v}_C \perp r_A \wedge \vec{v}_C \perp r_B \Rightarrow \vec{v}_C = 0$
 \Rightarrow ROTAZIONE (C PUÒ ESSERE ESTERNO ALLA FIGURA)



SE $\vec{v}_A \parallel \vec{v}_B$ E $|\vec{v}_A| = |\vec{v}_B|$, C DEGENERNA

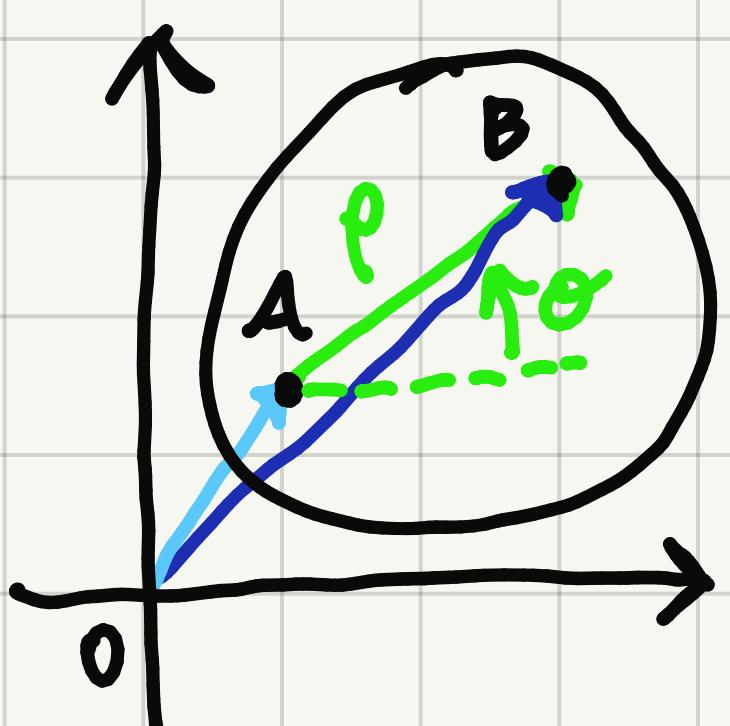
POLIFERI $r_A \parallel r_B \Rightarrow$ TRASLAZIONE



SE $\vec{v}_A \parallel -\vec{v}_B$ E $|\vec{v}_A| = |\vec{v}_B|$,

$\exists C = \Gamma_{AB} \cap \text{CONG}$

SPOSTAMENTO



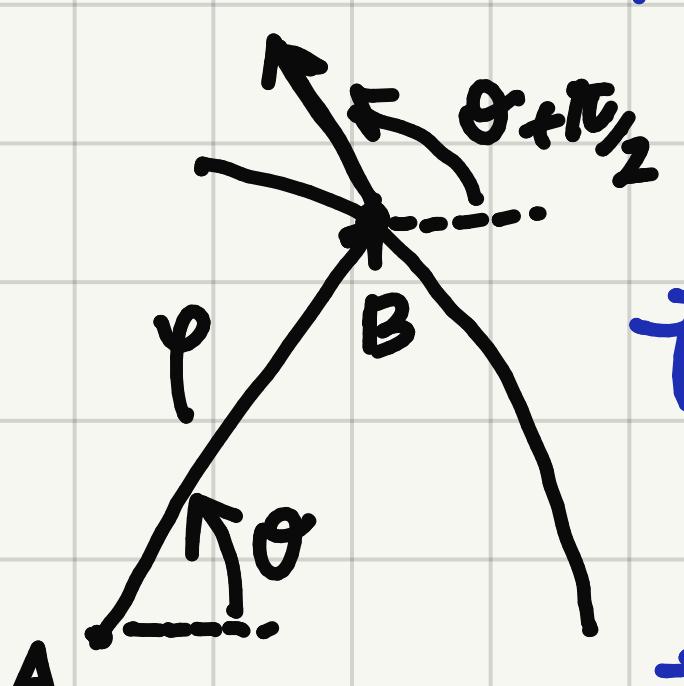
- SCEGLIO A DI CARATTERISTICHE NOTE
- DESCRIVO B CHE GENERALIZZA TUTTI I PUNTI APPARTENENTI AL CORPO RIGIDO

φ FISSO PER DEFINIZIONE CORPO RIGIDO

OBIEKTIVO

$$\vec{P}_B = (B-O) = (A-O) + (B-A) = \vec{r}_A + \rho e^{i\varphi}$$

VELOCITÀ



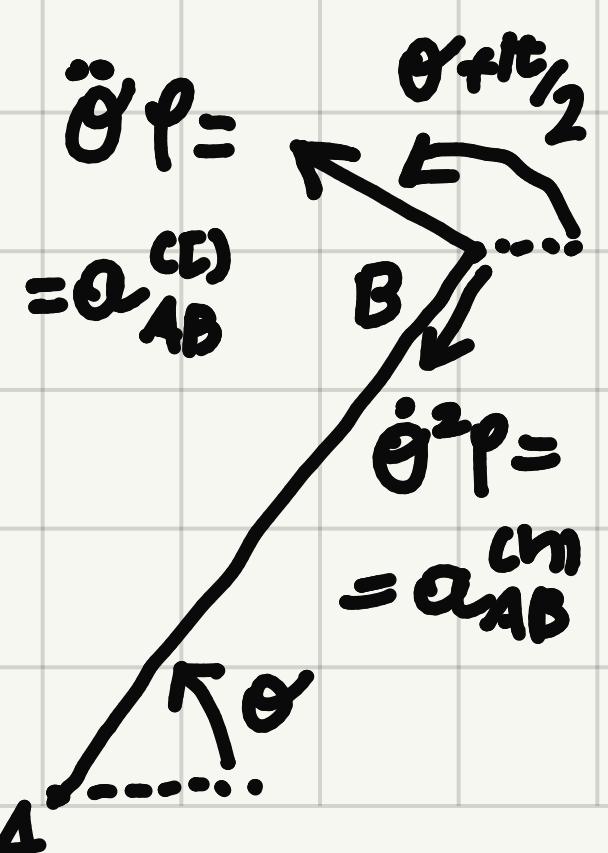
$$\vec{v}_B = \frac{d}{dt} \vec{P}_B = \vec{v}_A + \rho \frac{d}{dt} e^{i\varphi} = \vec{v}_A + \rho i e^{i\varphi} \dot{\varphi} =$$

$$= \vec{v}_A + \rho \dot{\varphi} e^{i(\varphi+\theta/2)}$$

= TRASLAZIONE A + ROTAZIONE BA

TEOREMA DI RIVALS

$$\vec{v}_B = \vec{v}_A + \vec{\omega} \times (\vec{CB} - \vec{CA})$$



$$\ddot{\vartheta} \rho = \vec{a}_A + \frac{d}{dt} \rho \dot{\varphi} e^{i(\varphi+\theta/2)} = \vec{a}_A + (\rho \ddot{\varphi} e^{i(\varphi+\theta/2)} -$$

! A ≈ CIR

$$\Rightarrow \vec{v}_A = 0$$

Moto ROTAZ.

$$\Rightarrow \vec{a}_A = 0$$

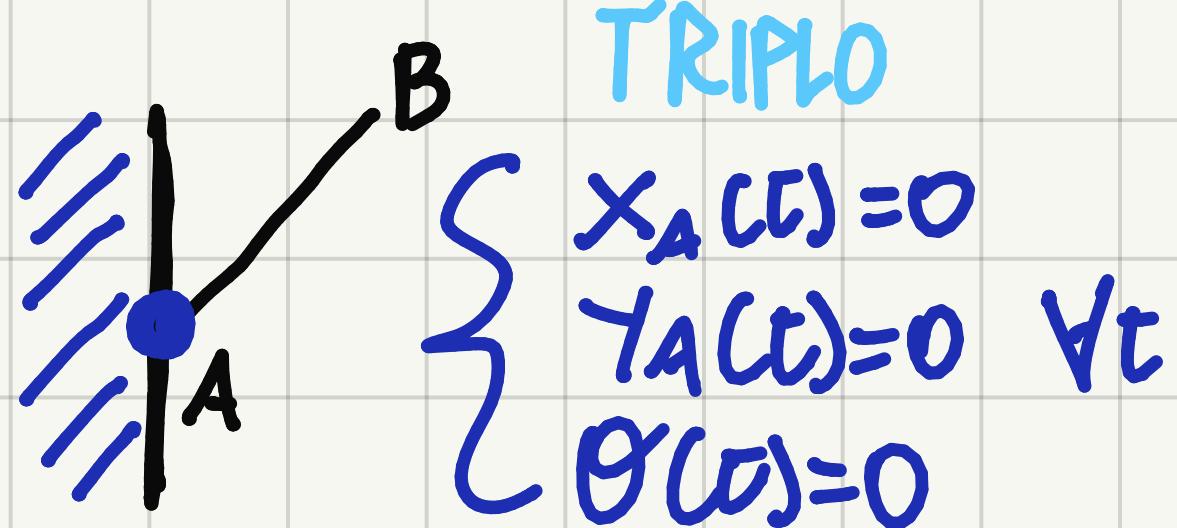
$$- \rho \dot{\varphi}^2 e^{i\varphi}) = \vec{a}_A + \vec{a}_{BA}^{(CIR)} + \vec{a}_{BA}^{(CIR)}$$

$$\vec{a}_B = \vec{a}_A + \vec{\omega} \times (\vec{CB} - \vec{CA}) + \vec{\omega}^2 \times (\vec{\omega} \times (\vec{CB} - \vec{CA}))$$

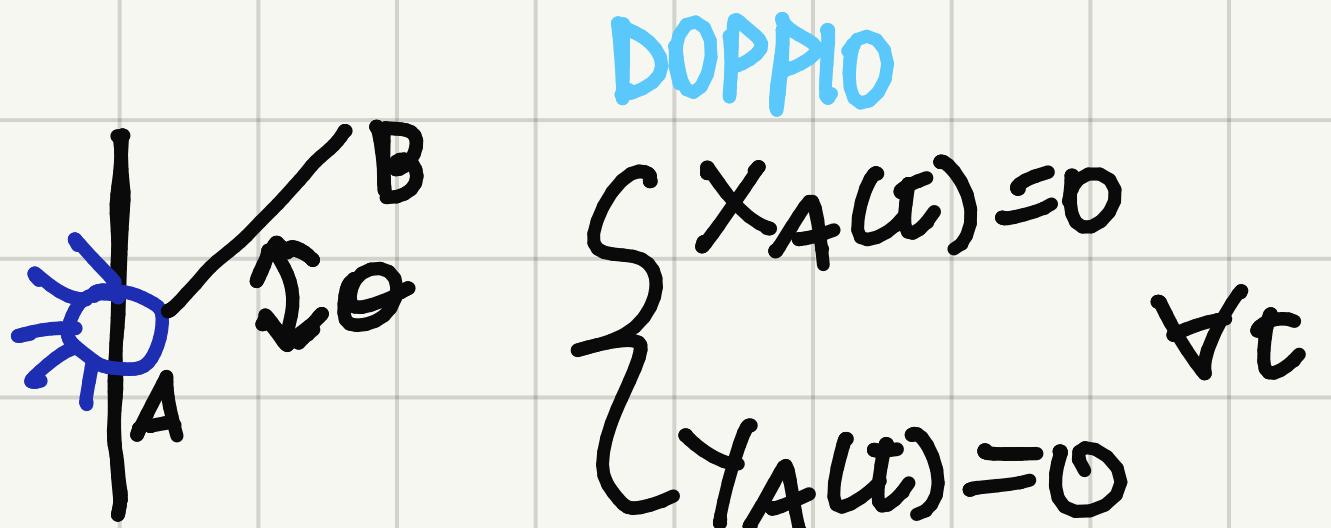
$$= \vec{a}_A + \vec{\omega} \times (\vec{CB} - \vec{CA}) - \omega^2 (\vec{CB} - \vec{CA}) \quad \text{NEL PIANO}$$

TIPI DI VINCOLI

VINCOLO DI INCASTRO

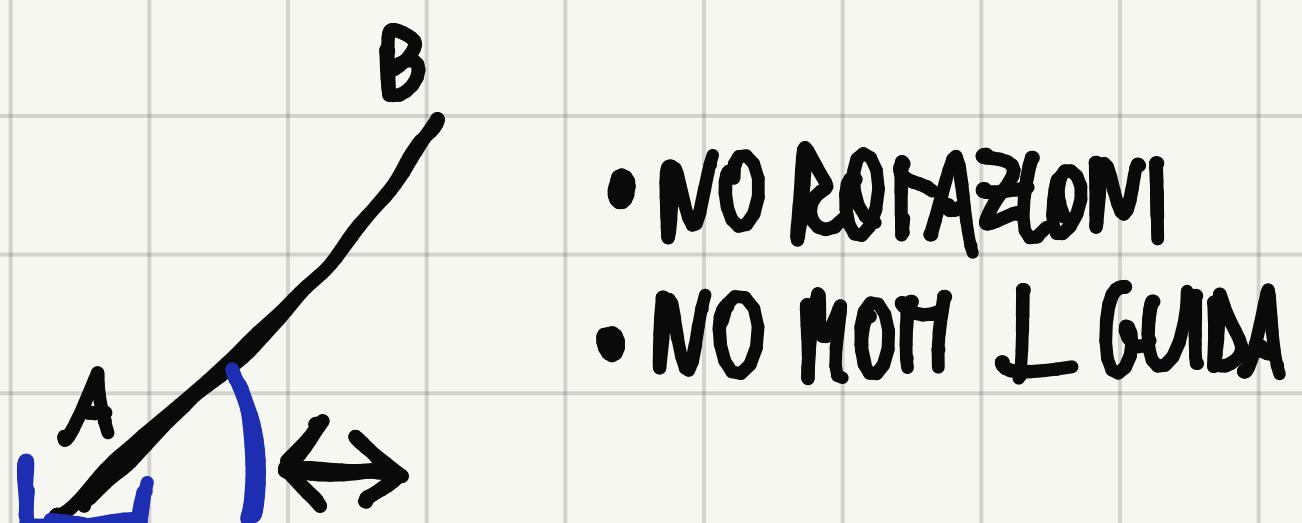


VINCOLO DI CERNIERA



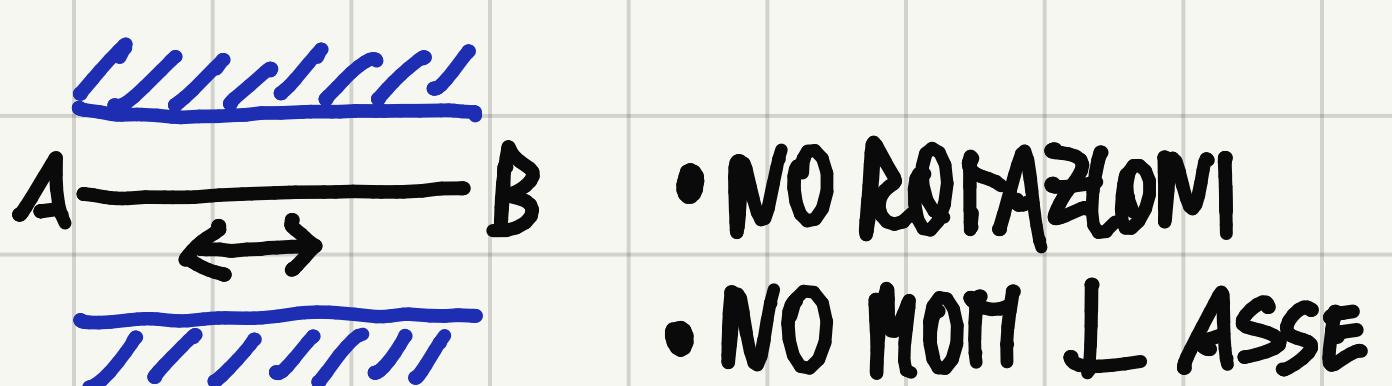
VINCOLO DI PATINO

DOPPIO



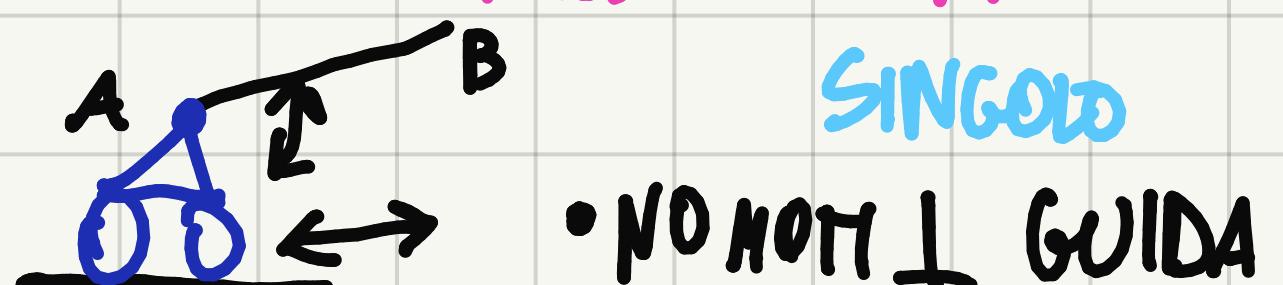
VINCOLO DI MANICOMO

DOPPIO



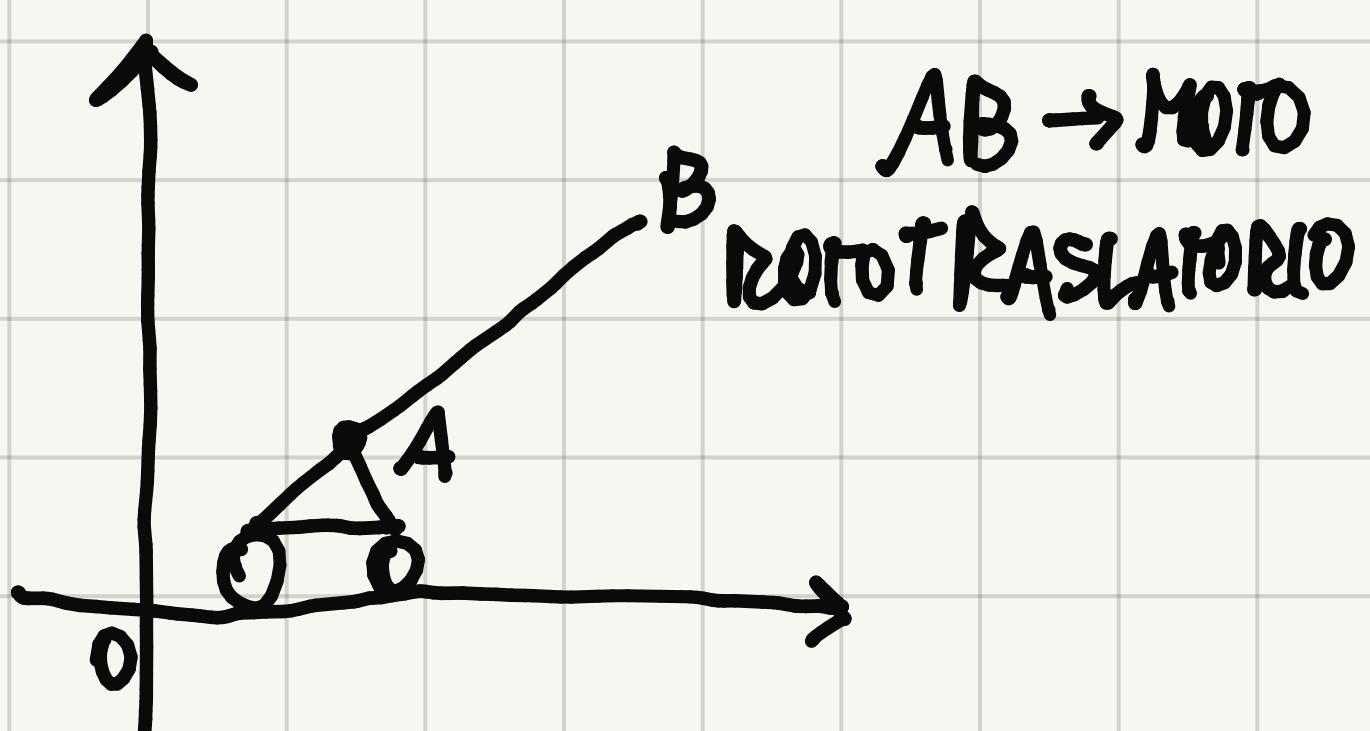
! IN 2D, PATINO = MANICOMO

VINCOLO CARRELLO + CERNIERA

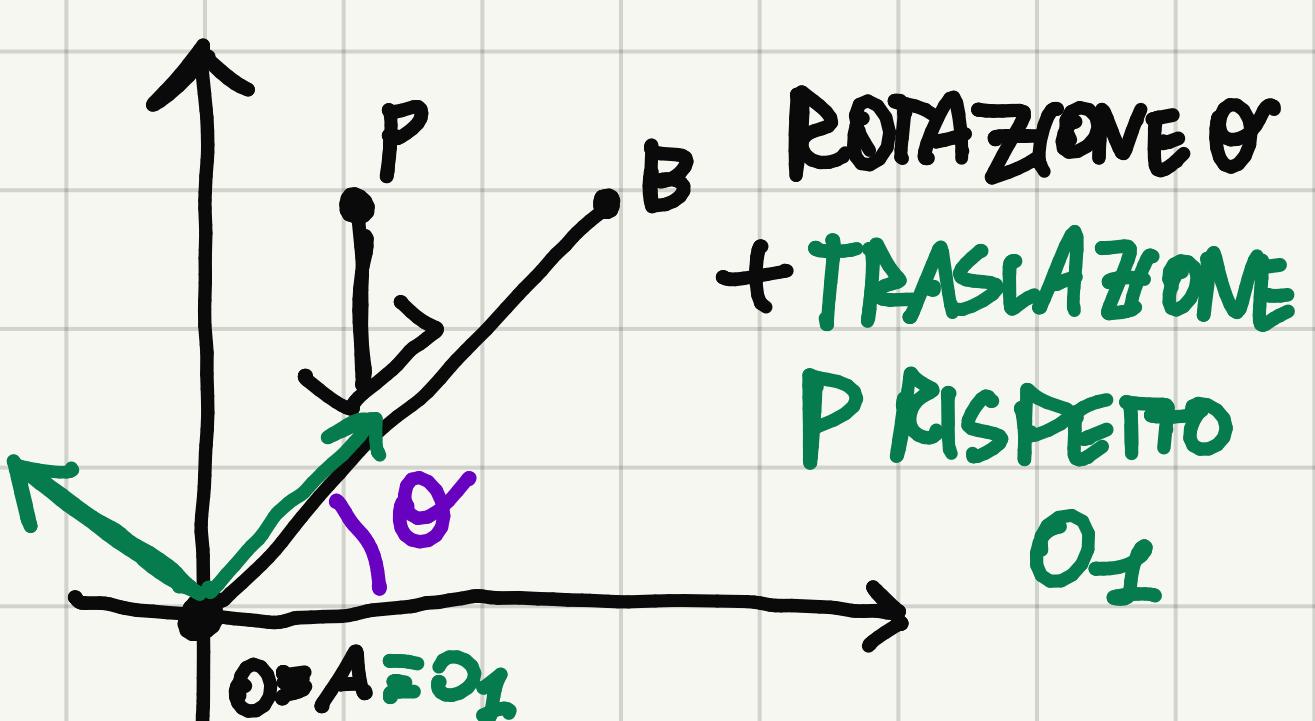
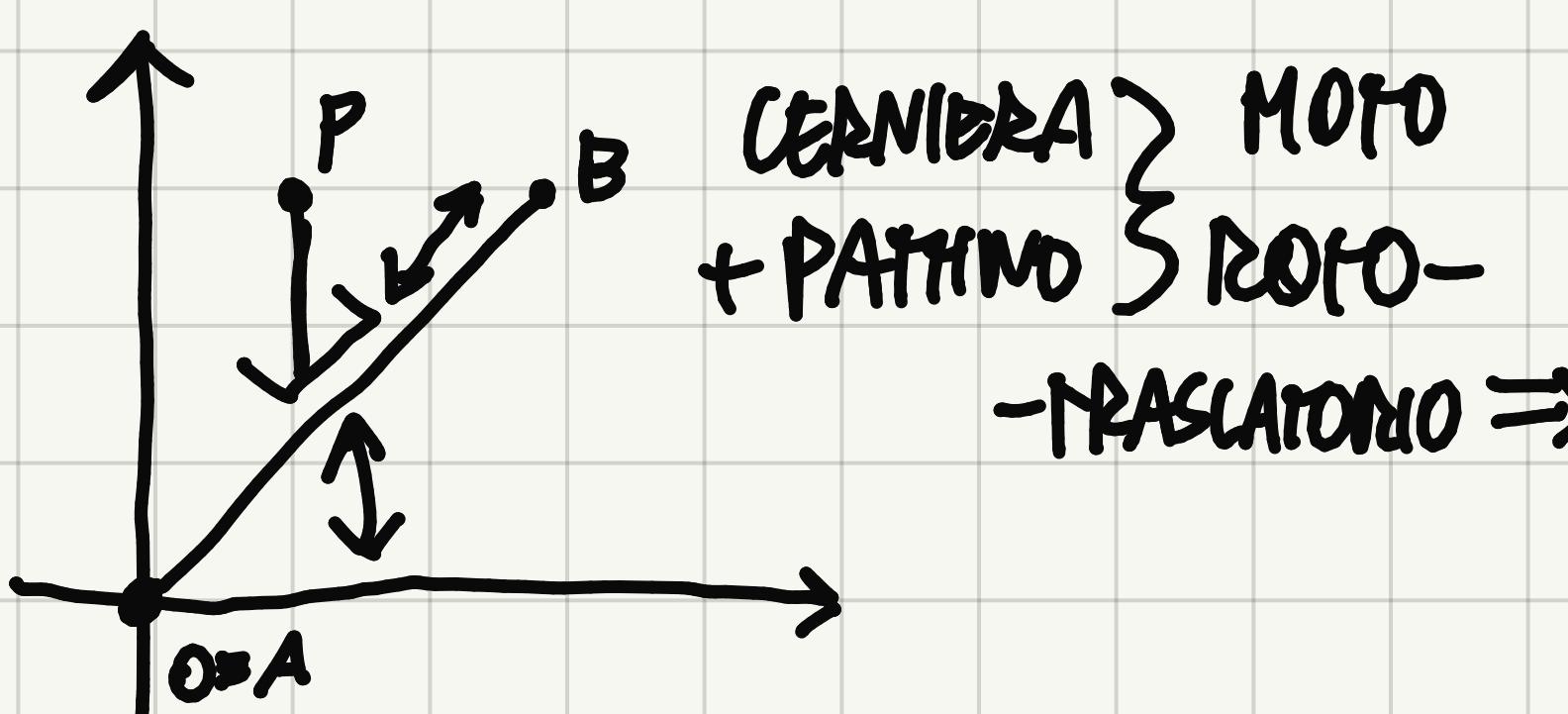
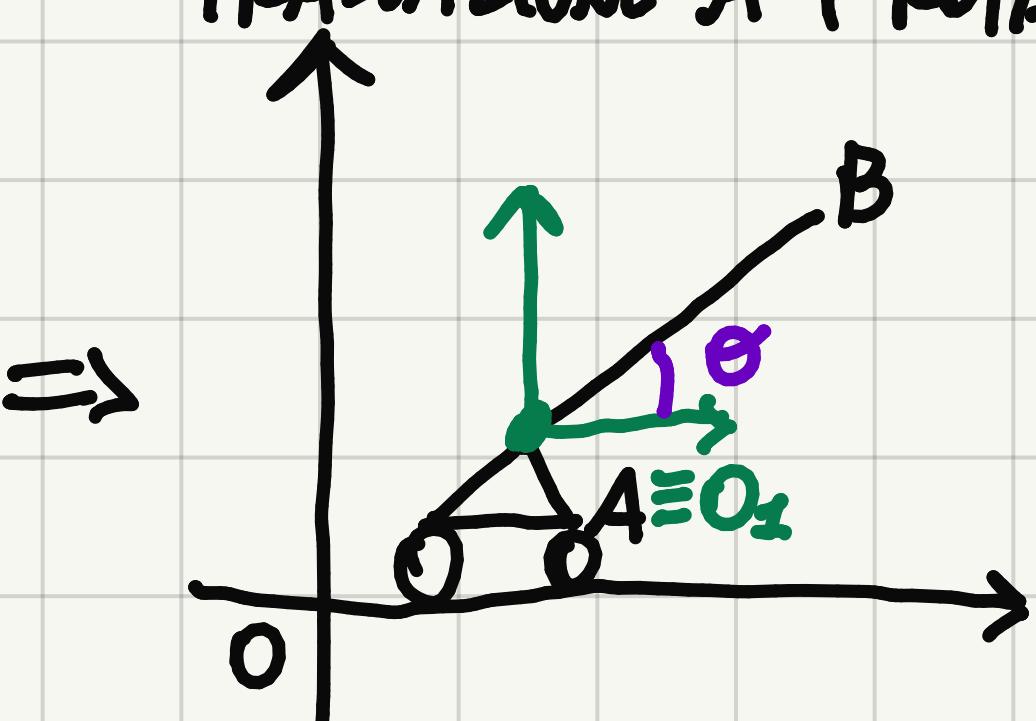


MOTI RELATIVI

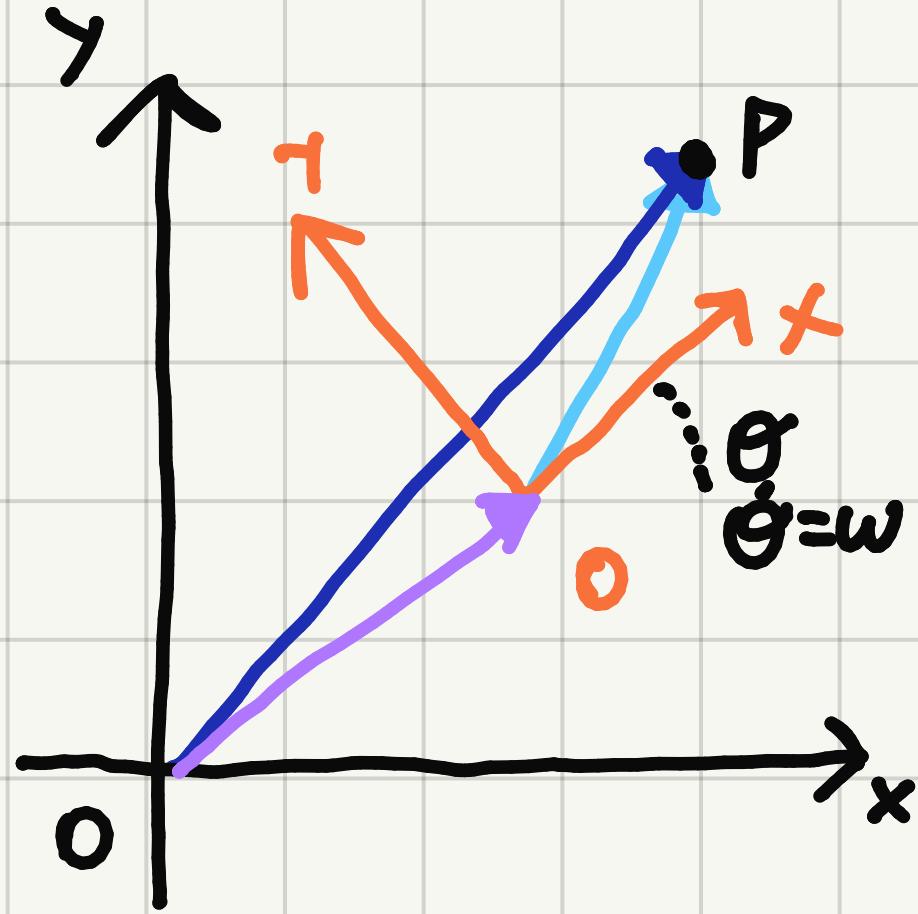
ESEMPI DI CASI UTILI



TRASLAZIONE A + ROTAZIONE θ



FORMULAZIONE



DI CARATTERISTICHE NOTE

! x, \dot{x} NON COSTANTI IN GENERALE
(SISTEMA IN ROTAZIONE)

$$(P-O) = (O-O) + (P-O)$$

$$\begin{cases} (P-O) = x_P \hat{x} + y_P \hat{z} & \text{OBIETTIVO} \\ (O-O) = x_O \hat{x} + y_O \hat{z} & \text{NOTO} \\ (P-O) = x_P \hat{x} + y_P \hat{z} & \text{DA CALCOLARE} \end{cases}$$

$$\vec{v}_P = \frac{d}{dt}(O-O) + \frac{d}{dt}(P-O) = \vec{v}_O + (\underbrace{\dot{x}_P \hat{x} + \dot{y}_P \hat{z}}_{\vec{v}_{relP}} + x_P \frac{d}{dt} \hat{x} + y_P \frac{d}{dt} \hat{z})$$

$\frac{d}{dt} \hat{x}$? ANALOGO PER \hat{z}

$$\begin{array}{l} \text{SCELGO } A \mid |(A-O)|=1 \Rightarrow \hat{x} = (A-O) = (A-O) - (O-O) \\ \text{VELOCITÀ} \\ \vec{v}_A = \vec{v}_O + \vec{v}_{relP} \\ \text{ACCELERAZIONE} \quad \vec{a}_A = \vec{a}_O + \vec{a}_{relP} \end{array}$$

$$\Rightarrow \frac{d}{dt} \hat{x} = \vec{v}_O + \vec{w} \times \hat{x} - \vec{v}_O = \vec{w} \times \hat{x}$$

EQUAZIONE DI PASCON

$$\vec{v}_P = \vec{v}_O + \vec{v}_{relP} + x_P \vec{w} \times \hat{x} + y_P \vec{w} \times \hat{z} = \vec{v}_O + \vec{v}_{relP} + \vec{w} \times (x_P \hat{x} + y_P \hat{z})$$

SI NOTI CHE, SE OXY NON RUOTA,
E COSTANTE $\Rightarrow w=0$

ACCELERAZIONE

$$\begin{aligned} \vec{a}_P &= \frac{d}{dt} \vec{v}_O + \frac{d}{dt} \vec{v}_{relP} + \frac{d}{dt} (\vec{w} \times (P-O)) = \vec{a}_O + \frac{d}{dt} (\dot{x}_P \hat{x} + \dot{y}_P \hat{z}) + \vec{w} \times (P-O) + \\ &+ \vec{w} \times \frac{d}{dt} (P-O) = \vec{a}_O + (\ddot{x}_P \hat{x} + \ddot{y}_P \hat{z} + \dot{x}_P \vec{w} \times \hat{x} + \dot{y}_P \vec{w} \times \hat{z}) + \\ &+ \vec{w} \times (\vec{v}_{relP} + \vec{w} \times (P-O)) = \vec{a}_O + \vec{a}_{relP} + \vec{w} \times \vec{v}_{relP} + \vec{w} \times \vec{v}_{relP} + \end{aligned}$$

$$+\vec{\omega} \times (\vec{\omega} \times (P-O)) = \vec{a}_c + \underbrace{\vec{\omega} \times (P-O)}_{\vec{a}_n} + \underbrace{\vec{\omega} \times (\vec{\omega} \times (P-O))}_{\vec{a}_{co}} + 2\vec{\omega} \times \vec{v}_{relip}$$

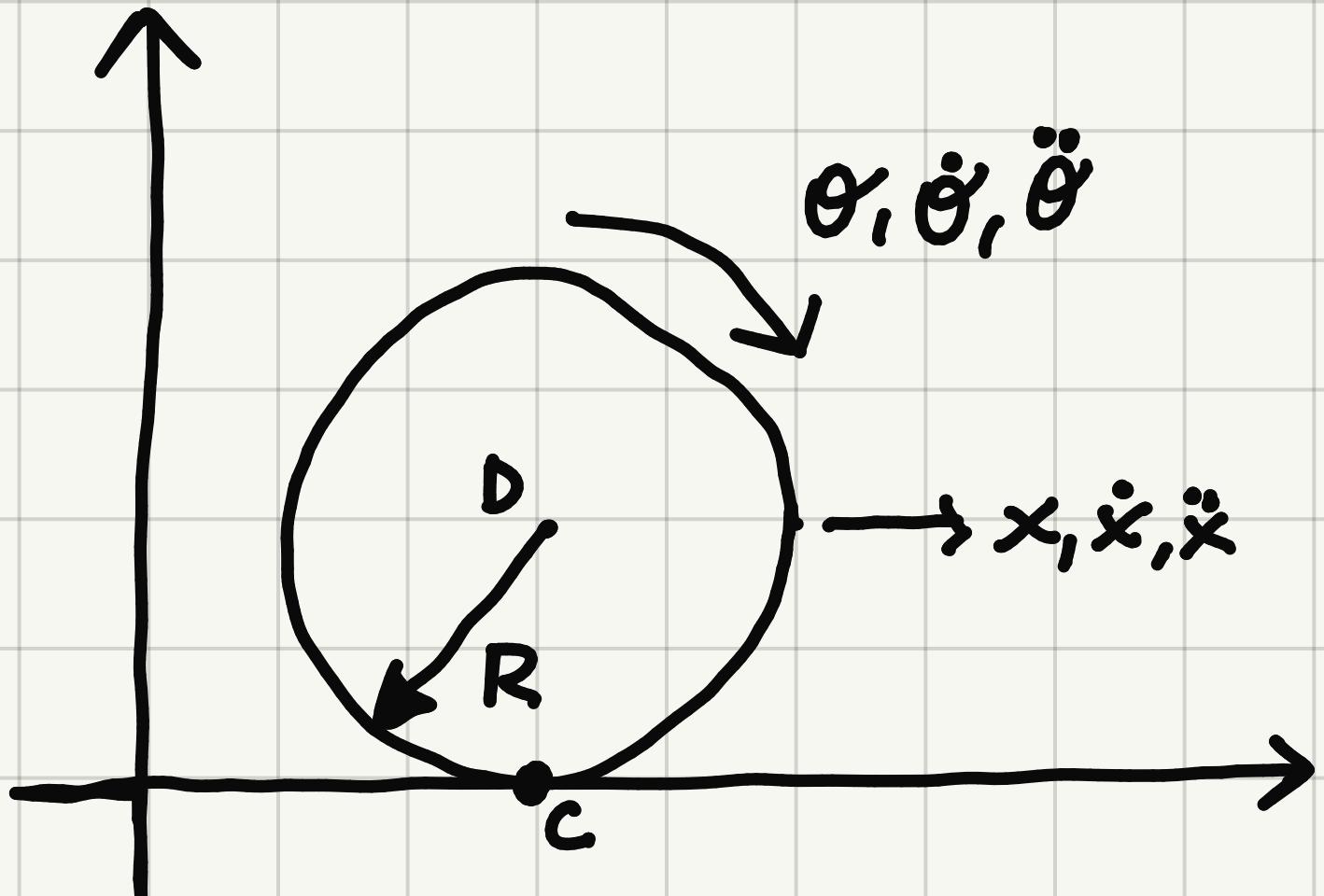
\vec{a}_c \vec{a}_n \vec{a}_{co}

\vec{a}_{rel}

! $\vec{a}_{co} = 0$ SE:

- TERRA TRASLANTE $\Rightarrow \vec{\omega} = 0$
- PELORO RIGIDO $\Rightarrow \vec{v}_{relip} = 0$
- $\vec{v}_{relip} \parallel \vec{\omega}$

VINCOLO DI ROTOLAMENTO



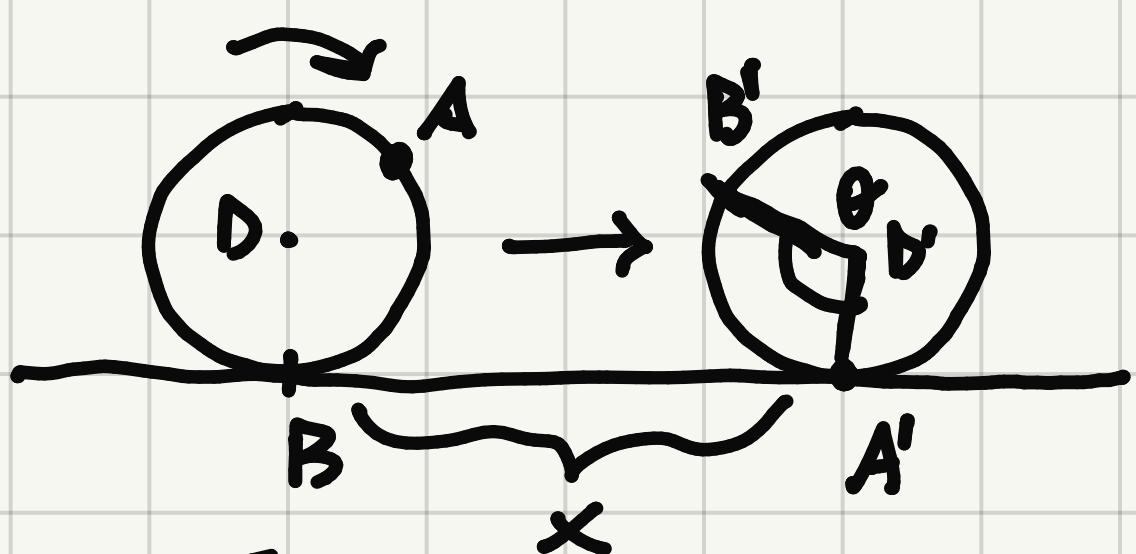
CASO VINCOLO DI CONTATTO

IMPEDISCO I MOTI LONGO LA \perp
ALLA GUIDA PIANA

$$\begin{cases} x_D = x \\ y_D = R \end{cases} \rightarrow \begin{cases} v_{x_D} = \dot{x} \\ v_{y_D} = 0 \end{cases} \rightarrow x, \theta \text{ INDEPENDENTI}$$

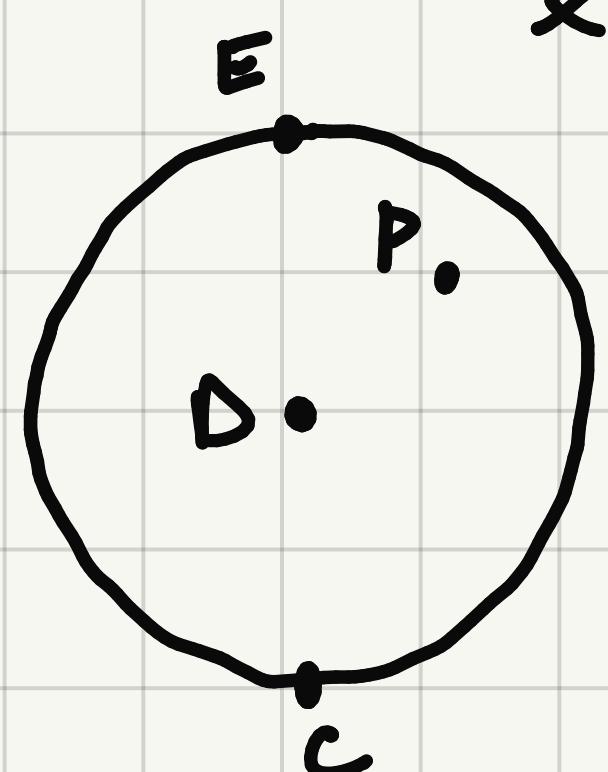
CASO VINCOLO DI PURO ROTOLAMENTO \rightarrow IN UNO SPECIFICO ESEMPI
 $\exists c \mid \vec{v}_c = 0$

IMPEDISCO TUTTE LE TRASLATORIE $\begin{cases} x_D = R \\ y_D = R \end{cases} \rightarrow \begin{cases} v_{x_D} = 0 \\ v_{y_D} = 0 \end{cases} \Rightarrow c \equiv \text{CIR} \quad x, \theta \text{ DIPENDENTI}$



$$DD' = x = A'B = R\theta \Rightarrow x = R\theta$$

$$\rightarrow \vec{v}_P = R\dot{\theta}\hat{x} = R\omega\hat{x} \quad \vec{a}_D = R\ddot{\theta}\hat{x}$$



PER UN GENERICO P, TEOREMA DI RENALS

$$\vec{v}_P = \vec{v}_c + \vec{\omega} \times (P-C) \quad \text{CASO } P \equiv C \rightarrow \vec{v}_P = R\omega\hat{x}$$

$$\text{CASO } P \equiv E \rightarrow \vec{a}_P = R\ddot{\theta}\hat{x}$$

$$\vec{a}_P = \vec{a}_c + \vec{\omega} \times (P-C) + \vec{\omega} \times (\vec{\omega} \times (P-C))$$

$$\vec{a}_c = \vec{a}_r + \vec{a}_n + \vec{a}_{co} = (R\ddot{\theta}\hat{x} - R\dot{\theta}\hat{x}) + R\dot{\theta}^2\hat{z} = R\dot{\theta}^2\hat{z}$$

CINEMATICA SISTEMA DI CORPI

① INDIVIDUARE I VINCOLI TRA I CORPI E CALCOLARE I G.D.L.

REGOLA DI GRÜBLER $\rightarrow n = n_o - n_v$

• $n_o = 3 \cdot \# \text{CORPI} (\text{ESCLUSO IL TELAIO})$

• $n_v = 1 \cdot \# \text{VINCOLI SINGOLI} + 2 \cdot \# \text{VINCOLI DOPPI} + 3 \cdot \# \text{VINCOLI TRIPPI}$

- $n < 0 (n_v > n_o) \rightarrow \text{STRUTTURA IPERSTATICA}$

- $n = 0 \rightarrow \text{STRUTTURA ISOSTATICA}$

- $n \geq 1 \rightarrow \text{MECCANISMO}$

NEI NOSTRI CASI
DI STUDIO, AVREMO
SEMPRE $n=1$

② CATENA CINEMATICA APERTA/CHIUSA

APERTA \rightarrow CIASCUN CORPO CHE COSTITUISCE IL SISTEMA, INCLUSO IL TELAIO, È

CONNESSO UNICAMENTE AL CORPO CHE LO PRECEDI E/O SEGUI

CHIUSA \rightarrow ALMENO UN CORPO È CONNESSO AD UN ALTRO CHE NB LO PRECEDI ME

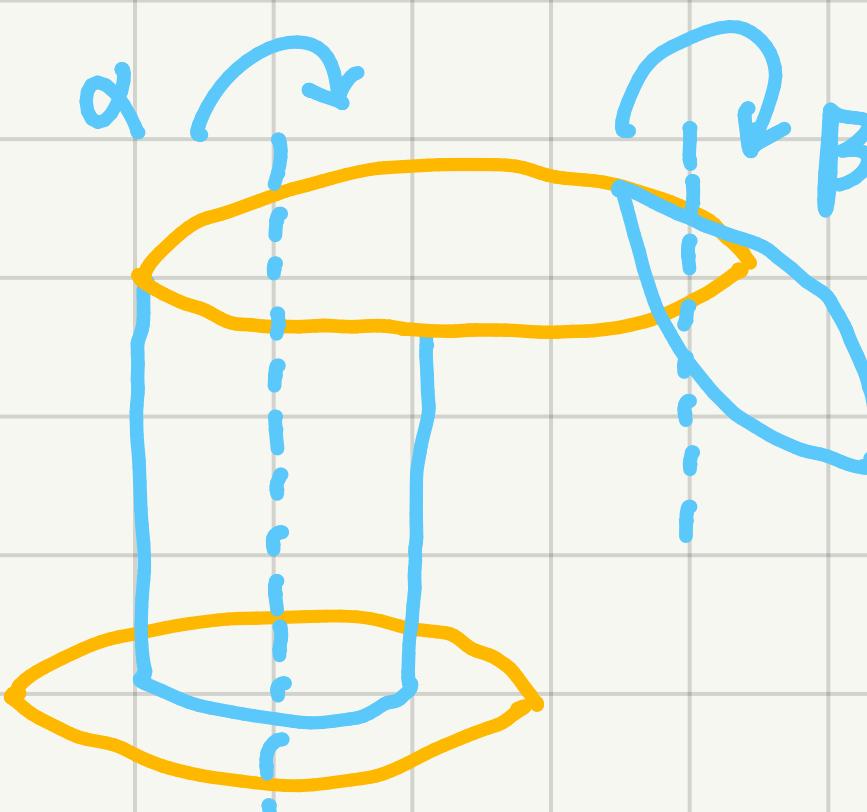
LO SEGUO

MANIPOLATORE/SCARABEO

SISTEMA A CATENA

APERTA CON 2

GRADI DI LIBERTÀ

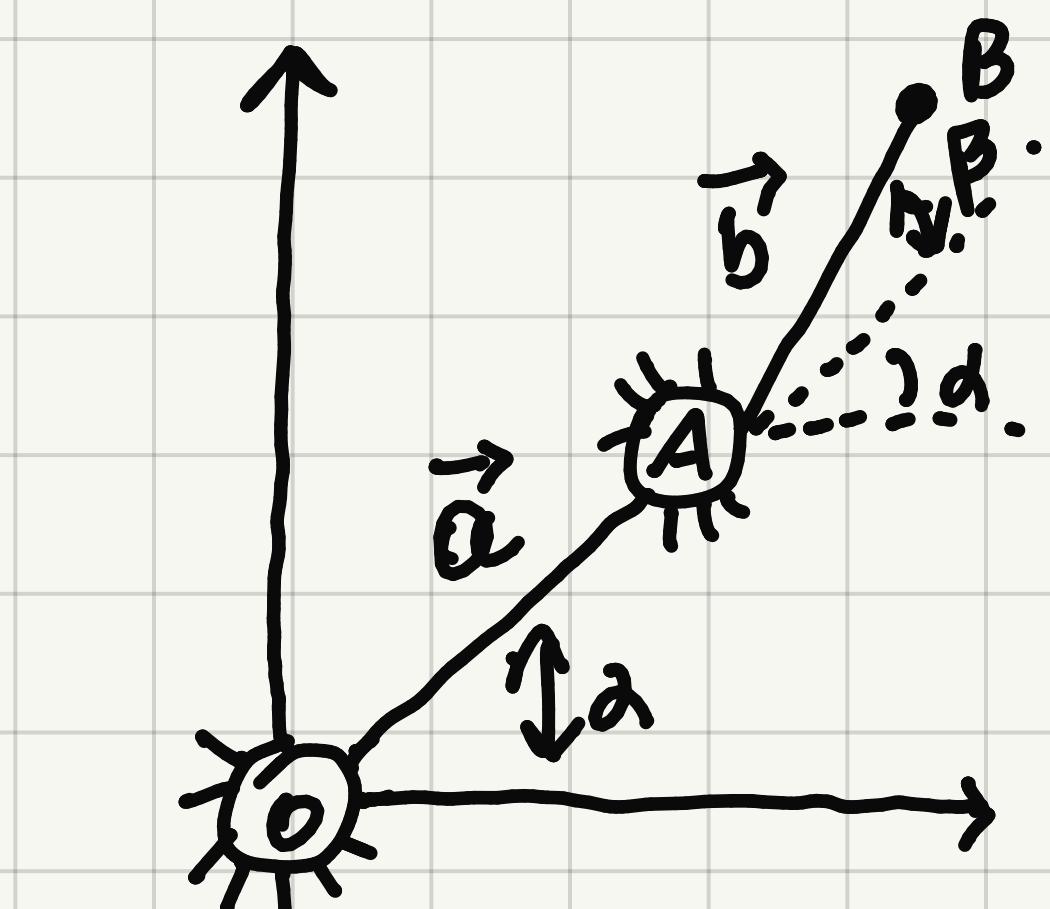


CERNIERE

MOTORI

che REGOLANO
ROTAZIONI

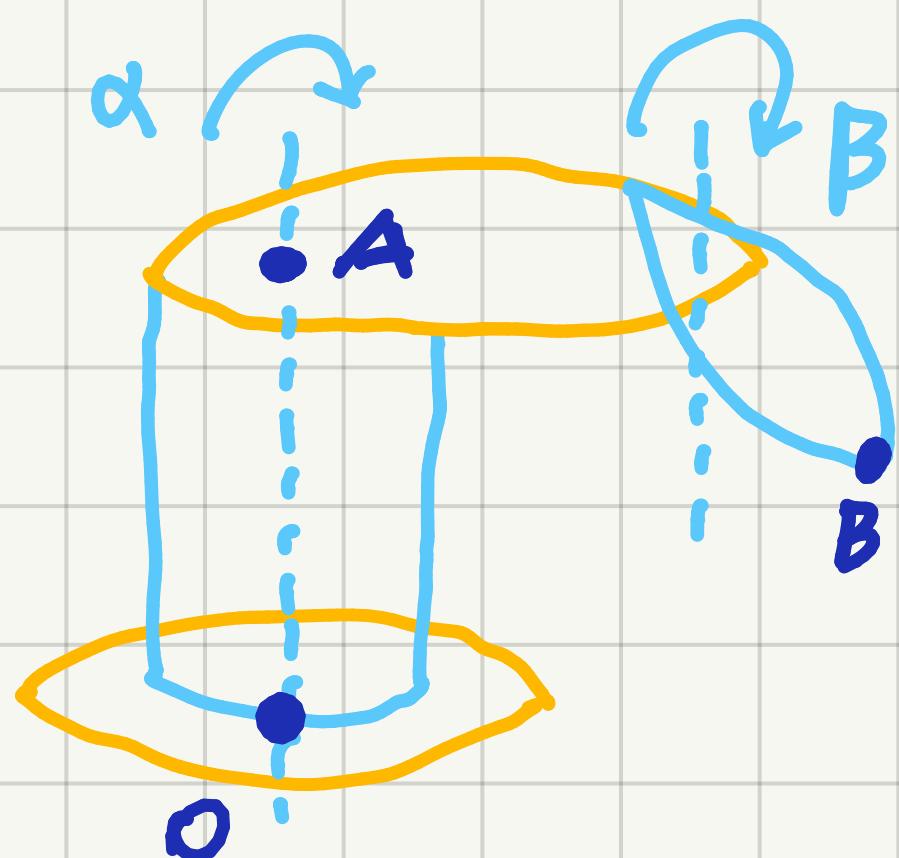
1 ANALISI DEL MOTORE CON EQUAZIONE VETTORIALI (LE DESCRIVE $\vec{CB}-\vec{O}$)



$$(\vec{B}-\vec{O}) = (\vec{A}-\vec{O}) + (\vec{B}-\vec{A})$$

* CERCHIO IN O E A

$$\begin{aligned} n &= n_0 - n_{\pi} = \\ &= 6 - 2 \cdot (2) = \\ &= 2 \end{aligned}$$



! $(\vec{B}-\vec{O})$ NON RAPPRESENTA NESSUN CORPO

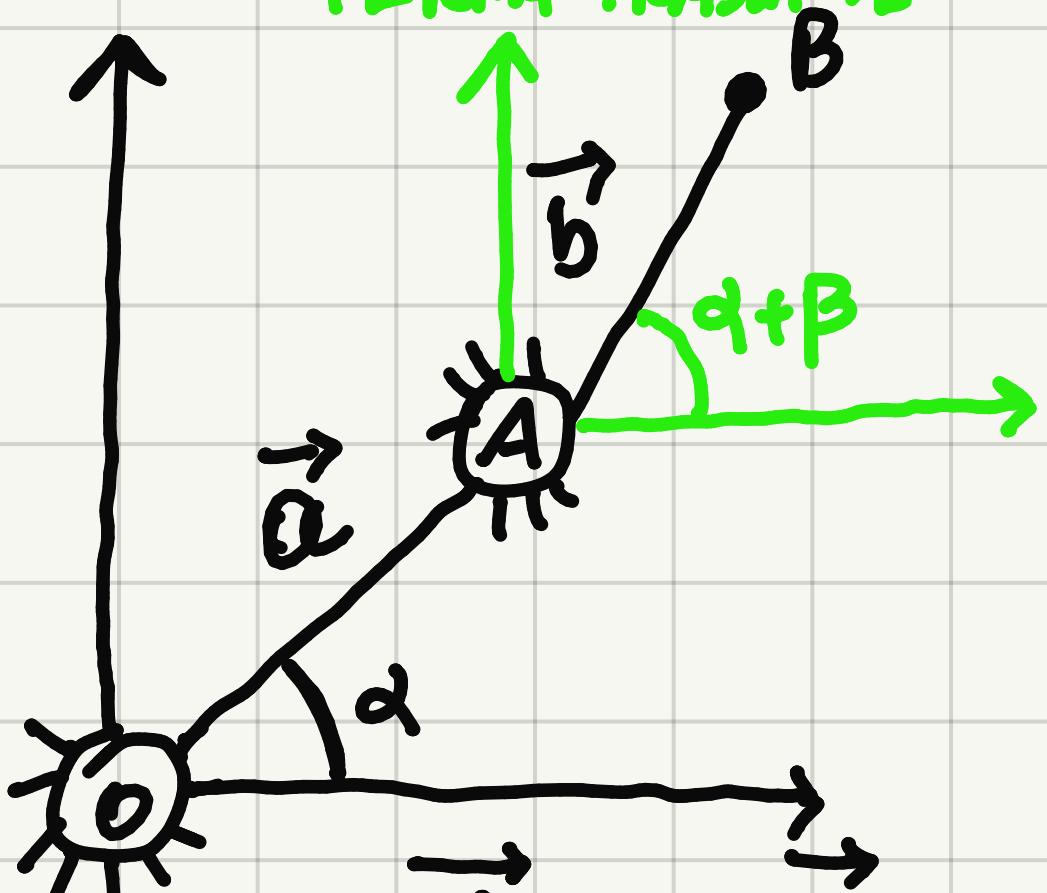
$$\vec{P} = a e^{i\alpha} + b e^{i(\alpha+\beta)}$$

$$\vec{v} = a \dot{a} e^{i(\alpha+\beta/2)} + b (\dot{a} + \dot{b}) e^{i(\alpha+\beta+\beta/2)}$$

$$\begin{aligned} \vec{a} = a \ddot{a} e^{i(\alpha+\beta/2)} - a \dot{a}^2 e^{i\alpha} + b (\ddot{a} + \ddot{b}) e^{i(\alpha+\beta+\beta/2)} - \\ - b (\dot{a} + \dot{b})^2 e^{i(\alpha+\beta)} \end{aligned}$$

2 MOTI RELATIVI

TERNA TRASLANTE



$$\vec{v} = \vec{v}_{crB} + \vec{v}_{relB}$$

$$\vec{v}_{crB} = \vec{v}_A + \vec{w} \times \vec{b} =$$

$$= \vec{w}_a \times \vec{a} = \dot{a} \hat{k} \times \vec{a} = a \dot{a} e^{i(\alpha+\beta/2)}$$

$$\vec{v}_{relB} = \vec{w}_{a+p} \times \vec{b} = (\dot{a} + \dot{b}) \hat{k} \times \vec{b} = b (\dot{a} + \dot{b}) e^{i(\alpha+\beta+\beta/2)}$$

$$\vec{\alpha} = \vec{\alpha}_{crB} + \vec{\alpha}_{relB} + \cancel{\vec{\alpha}_{co}}$$

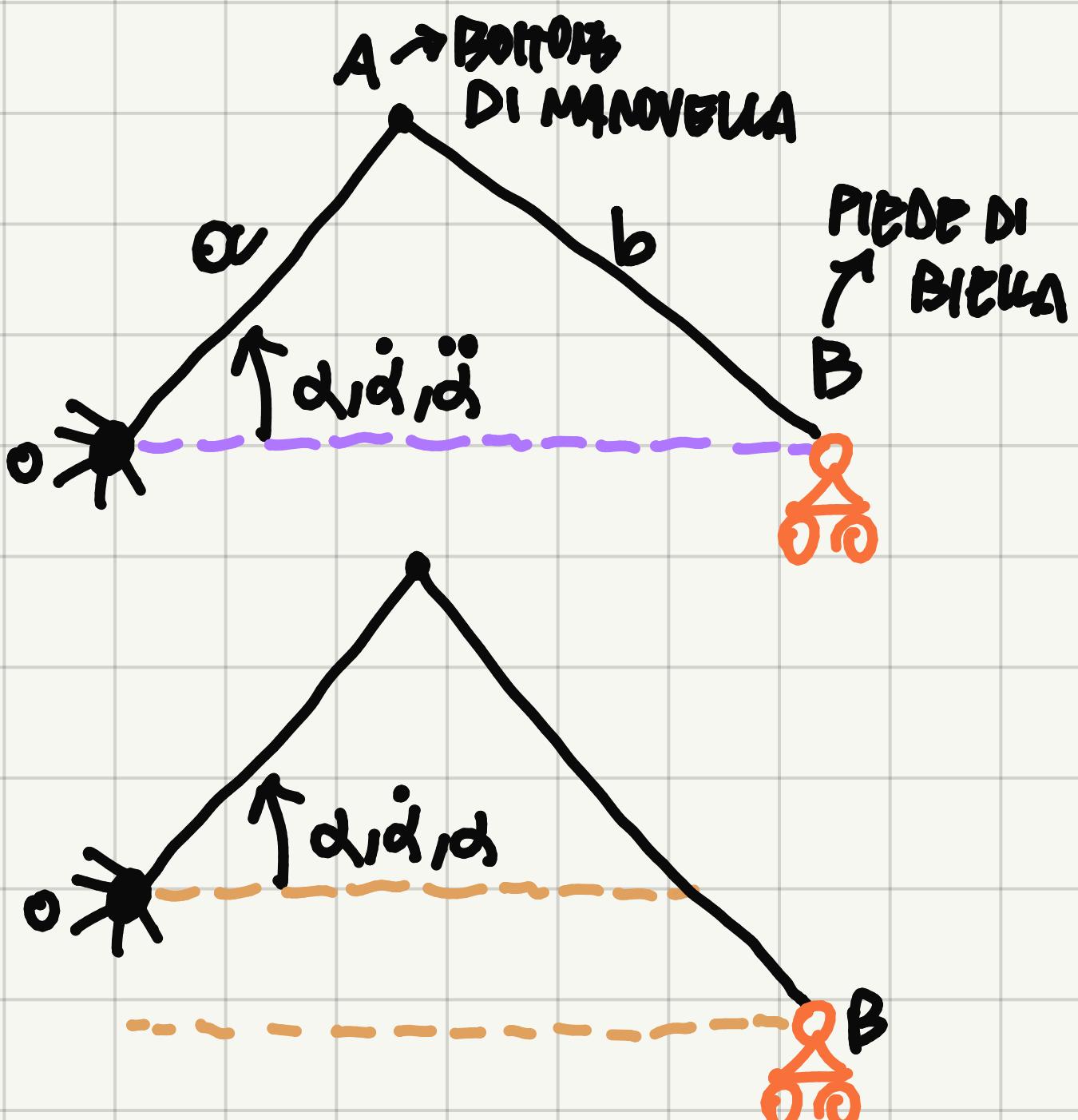
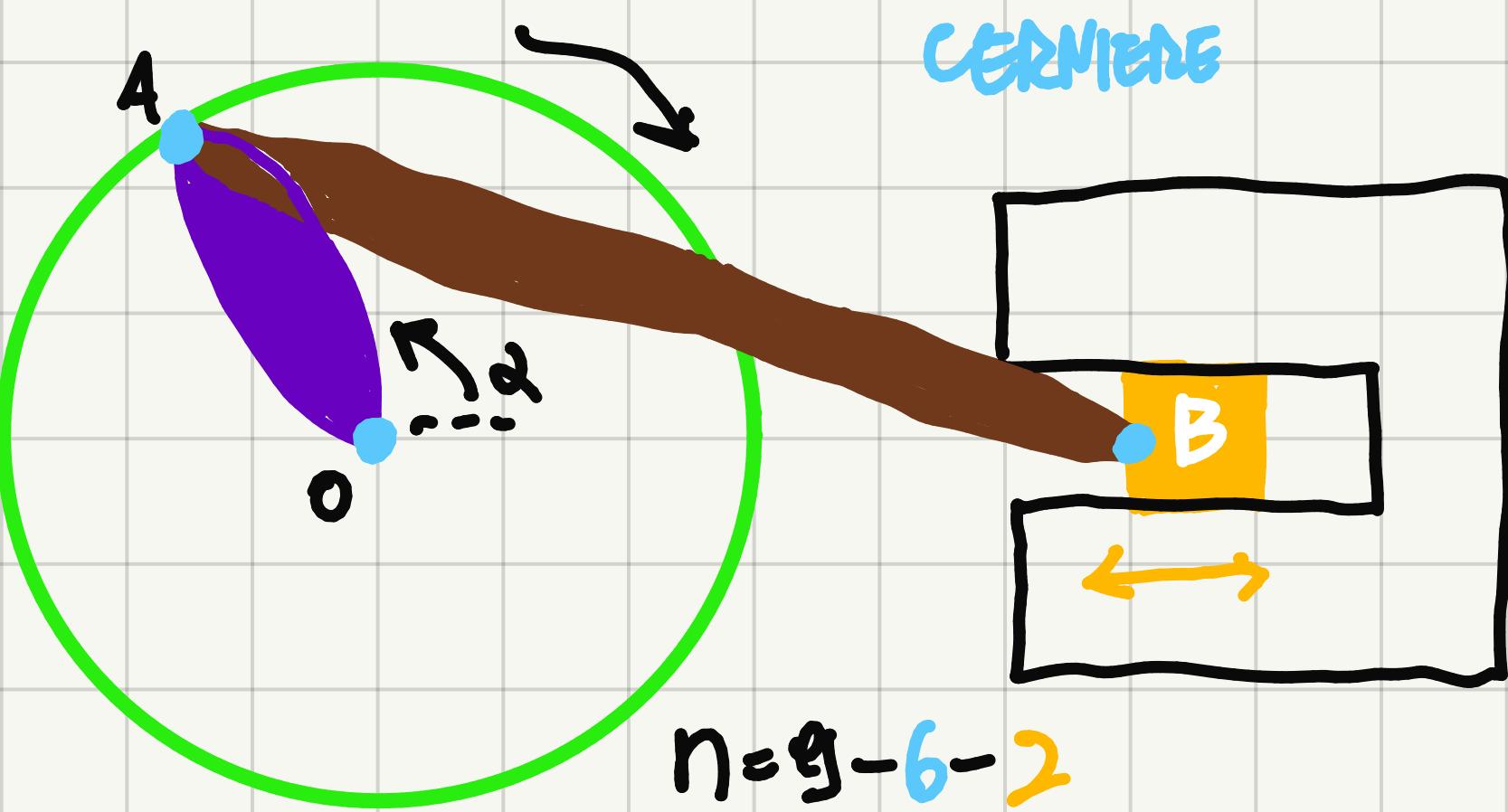
$$\vec{\alpha}_{crB} = \vec{\alpha}_A = a \ddot{a} e^{i(\alpha+\beta/2)} - a \dot{a}^2 e^{i\alpha}$$

$$- b (\dot{a} + \dot{b})^2 e^{i(\alpha+\beta)} -$$

$$\vec{\alpha}_{relB} = \vec{w}_{a+p} \times \vec{b} + \vec{w} \times (\vec{w} \times \vec{b}) = b (\ddot{a} + \ddot{b}) e^{i(\alpha+\beta+\beta/2)} -$$

MANOVELLISMO ORDINARIO

SISTEMA A
CATENA CHUSA
CON 1 GRADO
DI LIBERTÀ

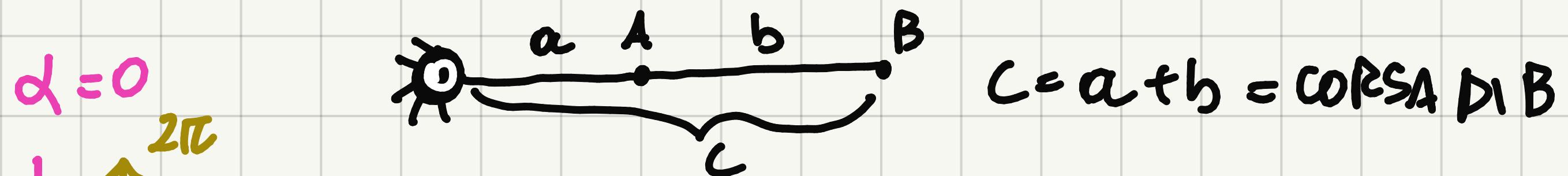


VINCOLO DI MANICOTTO APPROXIMATO
A CARRELLO

← MANOVELLISMO ORDINARIO
CENTRATO

← MANOVELLISMO ORDINARIO DEVIATO
(NON STUDIATO)

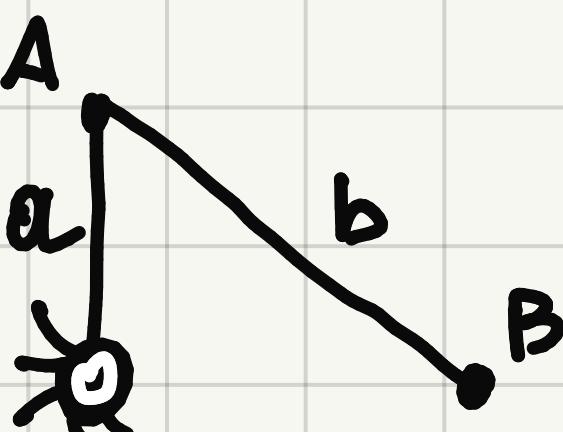
! B SI AVVICINA/ALLONTANA DA O IN FUNZIONE DI α



$$c = a + b = c_{\max}$$

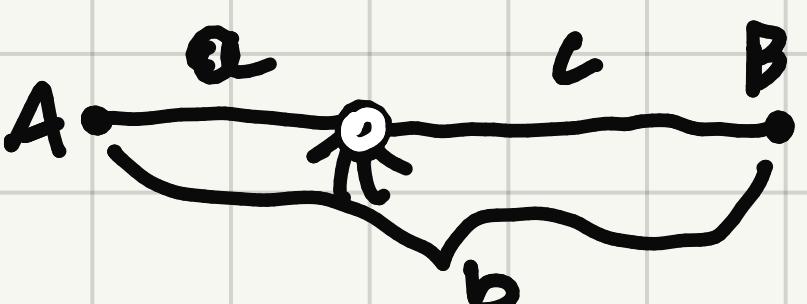
$$c = a + b = c_{\max} \quad v_B = 0, |\vec{a}_B| = |a_{\max}|$$

\Rightarrow PUNTO MORFO ESTERNO ($c_{\max}, v=0, a_{\max}$)



! ROMZIONE COMPLETA $\Leftrightarrow b > a$

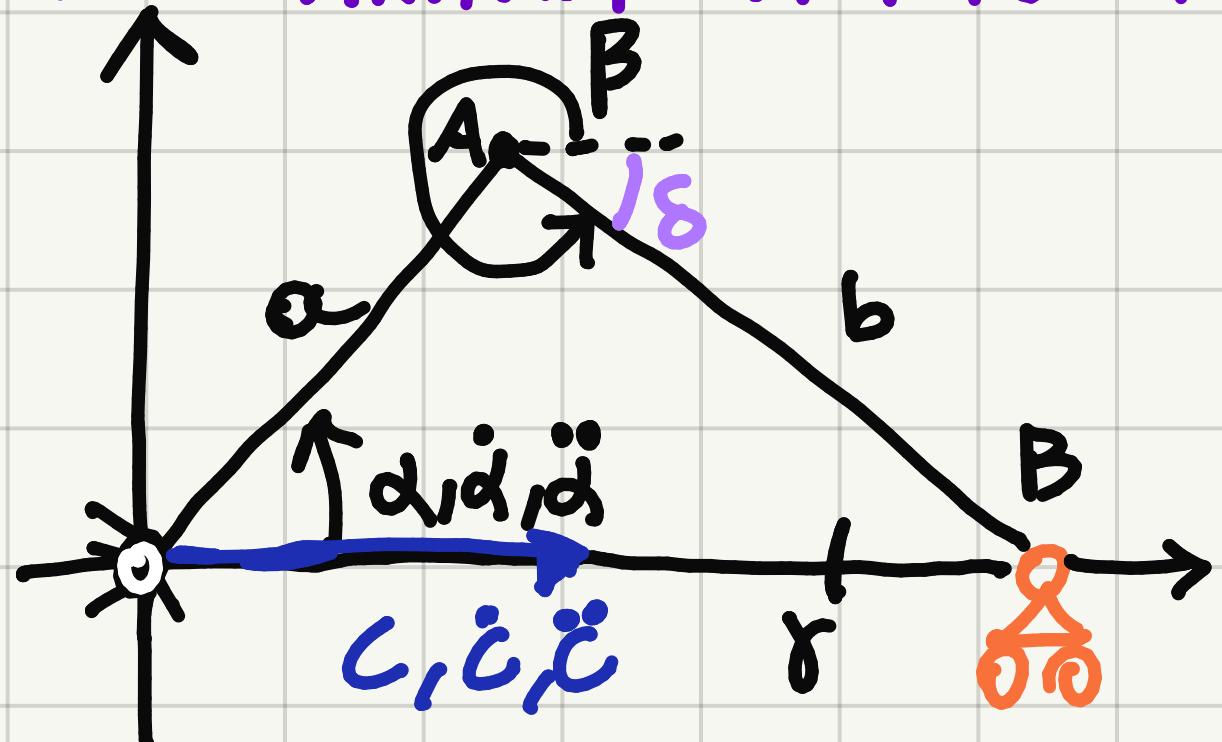
$|\vec{v}_B| \approx |v_{\max}|$ RAPPORTO CARATTERISTICO MANOVELLISMO $\lambda = \frac{a}{b} \ll 1$



$$c = b - a = c_{\min} \quad v_B = 0, |\vec{a}_B| = |a_{\max}|$$

\Rightarrow PUNTO MORFO INTERNO ($c_{\min}, v=0, a_{\max}$)

1 ANALISI DEL MOTORE CON EQUAZIONE VETTORIALE (LE DESCRITTE $\vec{CB} = \vec{O}$)



α NON COSTANTE
 β NON COSTANTE
 $\delta = 0$

$$\vec{C} = \vec{a} + \vec{b} \rightarrow C e^{i\delta} = a e^{i\alpha} + b e^{i\beta}$$

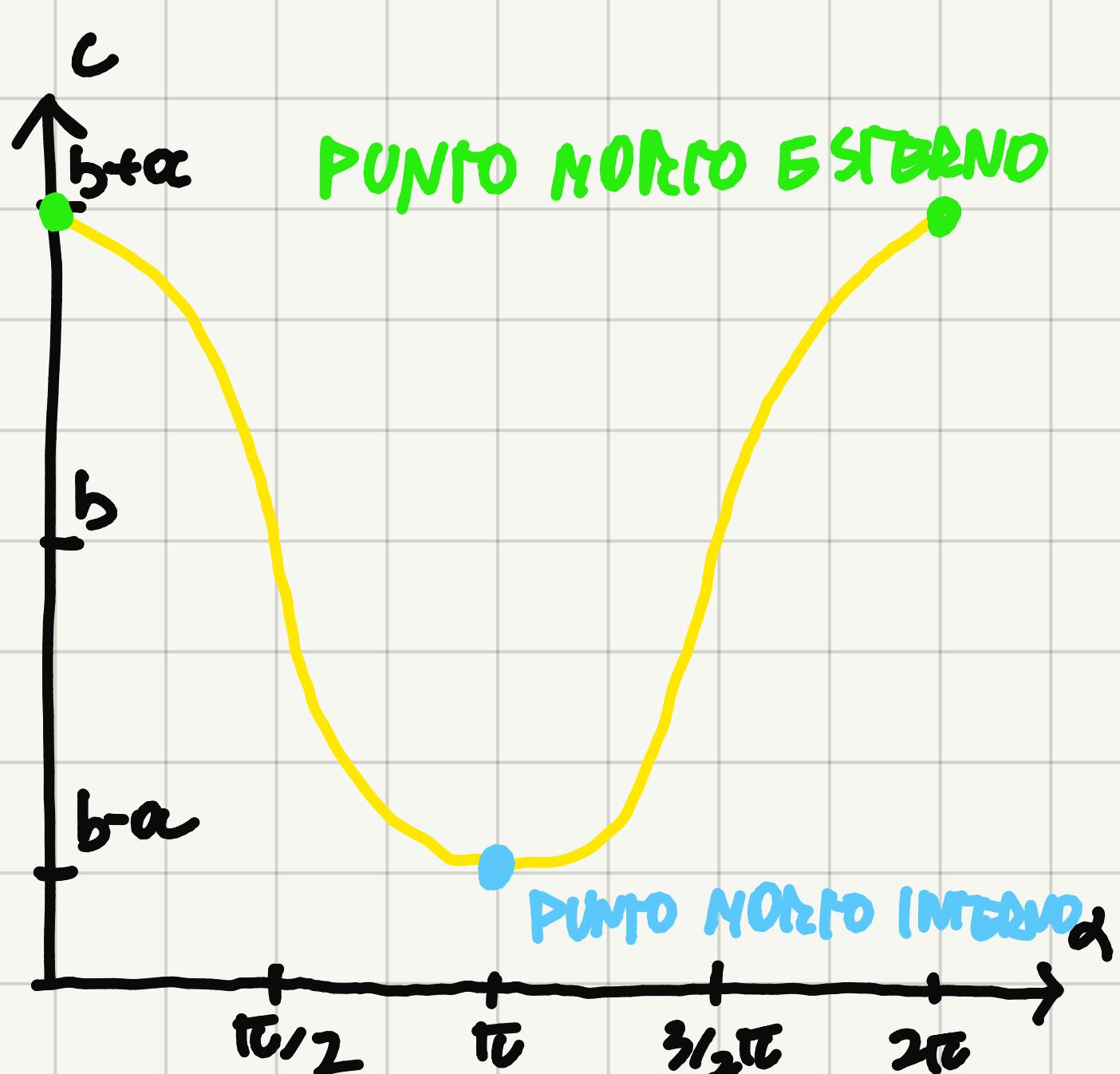
$$C = (a \cos \alpha + b \cos \beta) + i(a \sin \alpha + b \sin \beta)$$

$$a \sin \alpha + b \sin \beta = 0 \Rightarrow \tan \beta = -\frac{a}{b} \sin \alpha = -\alpha \sin \alpha$$

$$\sin^2 \beta + \cos^2 \beta = 1 \Rightarrow \cos \beta = \pm \sqrt{1 - \alpha^2 \sin^2 \alpha} = \sqrt{1 - \alpha^2 \sin^2 \alpha}$$

$$\cos \beta = \cos \delta, -\pi/2 \leq \delta \leq \pi/2 \Rightarrow \cos \delta = \cos \beta > 0$$

$$\Rightarrow C = a \cos \alpha + b \sqrt{1 - \alpha^2 \sin^2 \alpha}$$



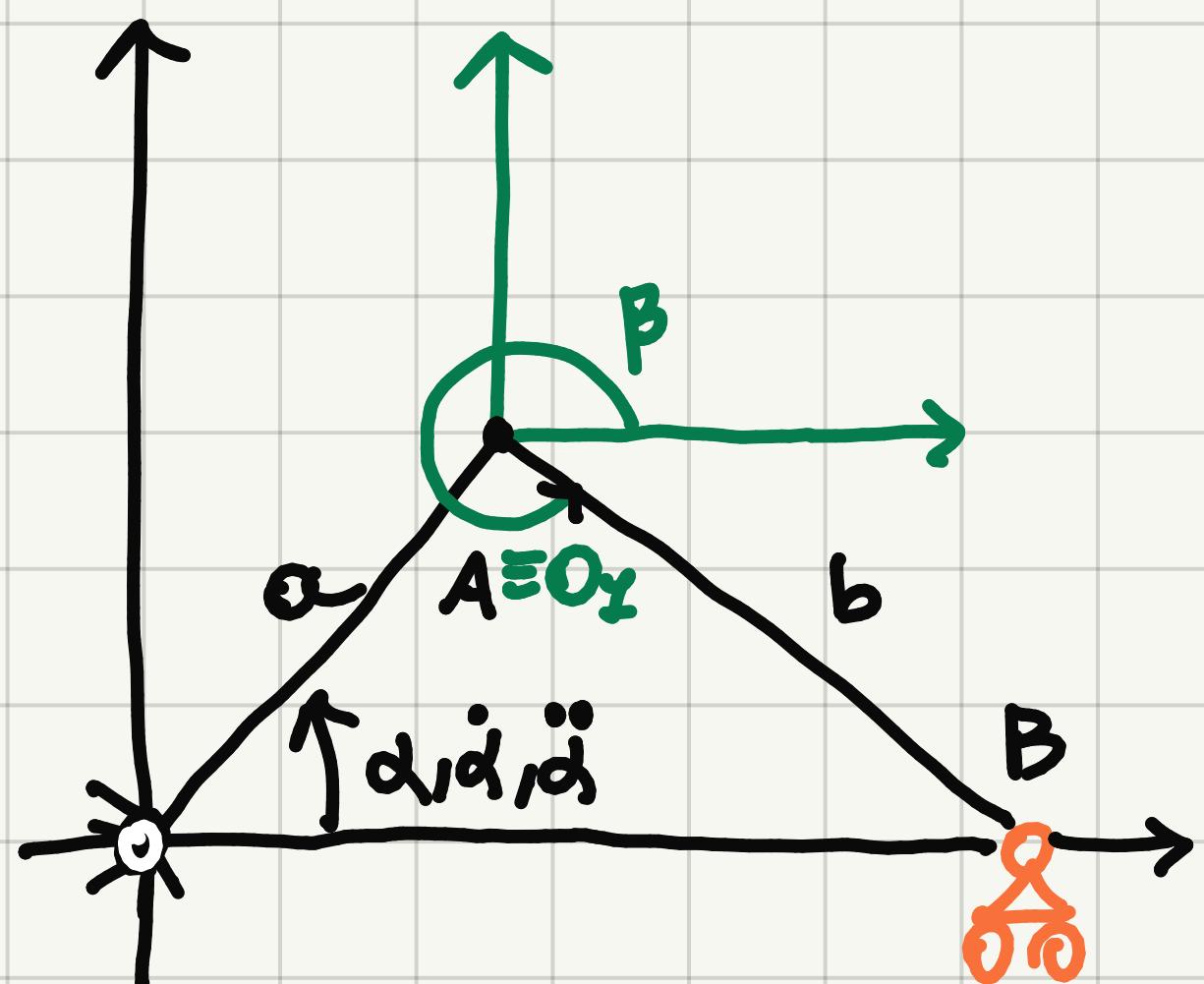
APPROXIMAZIONE
 DEL PRIMO ORDINE:
 $b \gg a \Rightarrow \alpha \rightarrow 0$
 $\Rightarrow C = a \cos \alpha + b$

$$\dot{C} = a \dot{\alpha} e^{i(\alpha + \pi/2)} + b \dot{\beta} e^{i(\beta + \pi/2)}$$

$$\ddot{C} = a \ddot{\alpha} e^{i(\alpha + \pi/2)} - a \dot{\alpha}^2 e^{i\alpha}$$

$$+ b \ddot{\beta} e^{i(\beta + \pi/2)} - b \dot{\beta}^2 e^{i\beta}$$

2 MOTI RELATIVI



TERNA TRASLANTE IN A

$$\vec{v}_B = \vec{v}_{tr_B} + \vec{v}_{rel_B}$$

$$= \alpha \dot{a} e^{i(\alpha + \pi/2)} + b \dot{\beta} e^{i(\beta + \pi/2)}$$

$$\vec{v}_B = v_B \hat{z}$$

$$\begin{cases} -\alpha \dot{a} \sin \alpha - b \dot{\beta} \sin \beta = \dot{c} \\ \alpha \dot{a} \cos \alpha + b \dot{\beta} \cos \beta = 0 \end{cases}$$

$$\begin{vmatrix} 1 & b \sin \beta \\ 0 & -b \cos \beta \end{vmatrix} \cdot \begin{vmatrix} \dot{c} \\ \dot{\beta} \end{vmatrix} = \begin{vmatrix} -\alpha \dot{a} \sin \alpha \\ \alpha \dot{a} \cos \alpha \end{vmatrix}$$

$$\Rightarrow \dot{\beta} = -\dot{a} \frac{a \cos \alpha}{b \cos \beta}, \quad \dot{c} = -\alpha \dot{a} (\sin \alpha - \cos \alpha \tan \beta)$$

$$\begin{aligned} \vec{a}_B &= a_{tr_B}^{(cc)} + a_{tr_B}^{(cn)} + a_{rel_B}^{(cc)} + a_{rel_B}^{(cn)} + \vec{a}_{co} \\ &= a \ddot{a} e^{i(\alpha + \pi/2)} - a \dot{a}^2 e^{ia} + b \ddot{\beta} e^{i(\beta + \pi/2)} - b \dot{\beta}^2 e^{i\beta} \end{aligned}$$

$$\begin{cases} -\alpha \ddot{a} \sin \alpha - \alpha \dot{a}^2 \cos \alpha - b \ddot{\beta} \sin \beta - b \dot{\beta}^2 \cos \beta = \ddot{c} \\ \alpha \ddot{a} \cos \alpha - \alpha \dot{a}^2 \sin \alpha + b \ddot{\beta} \cos \beta - b \dot{\beta}^2 \sin \beta = 0 \end{cases}$$

$$\begin{vmatrix} 1 & b \sin \beta \\ 0 & -b \cos \beta \end{vmatrix} \cdot \begin{vmatrix} \ddot{c} \\ \ddot{\beta} \end{vmatrix} = \begin{vmatrix} -\alpha \ddot{a} \sin \alpha - \alpha \dot{a}^2 \cos \alpha - b \dot{\beta}^2 \cos \beta \\ \alpha \ddot{a} \cos \alpha - \alpha \dot{a}^2 \sin \alpha - b \dot{\beta}^2 \sin \beta \end{vmatrix}$$

3 JACOBIANO DEL MOTORE

$$\Lambda = \frac{v_B}{\dot{\alpha}} = \frac{\dot{c}}{\dot{\alpha}}, \text{ LEGAME CINEMATICO TRA LA VELOCITÀ DEL GRADO DI LIBERTÀ USATO PER DESCRIVERE IL SISTEMA } (\dot{\alpha}) \text{ E LA VELOCITÀ DEL PUNTO}$$

DI INTERESSE B (C)

$$\dot{c} = \Lambda \dot{\alpha} \quad \Lambda = \frac{\dot{c}}{\dot{\alpha}} = \frac{dc}{d\alpha} \cdot \frac{d\alpha}{d\dot{\alpha}} = \frac{dc}{d\alpha} : \frac{d\alpha}{d\dot{\alpha}} = \frac{dc}{d\dot{\alpha}}$$

$$\dot{c} = -\alpha \dot{\alpha} (\sin \alpha - \cos \alpha \tan \beta)$$

$$\Rightarrow \Lambda(\alpha) = -\alpha (\sin \alpha - \cos \alpha \tan \beta)$$

$$= -\alpha (\sin \alpha - \cos \alpha (\arccos \left(-\frac{\alpha}{b} \sin \alpha \right)))$$

$$\ddot{c} = \frac{d\Lambda(\alpha)}{d\alpha} \dot{\alpha} + \Lambda(\alpha) \ddot{\alpha} = \frac{d^2 c}{d\alpha^2} \dot{\alpha}^2 + \frac{dc}{d\alpha} \dot{\alpha}$$