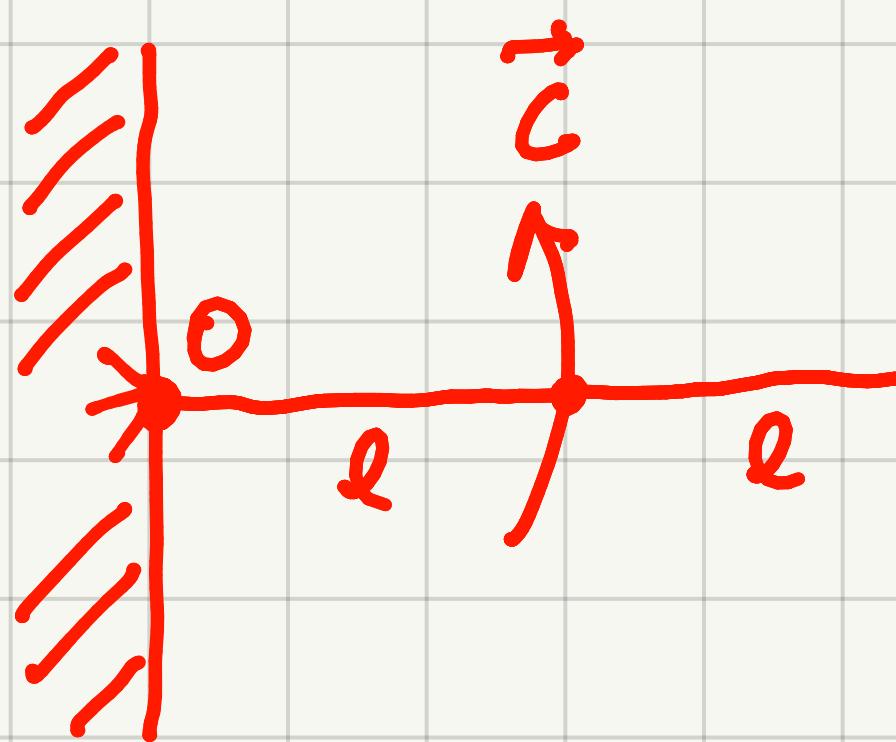


STATIC

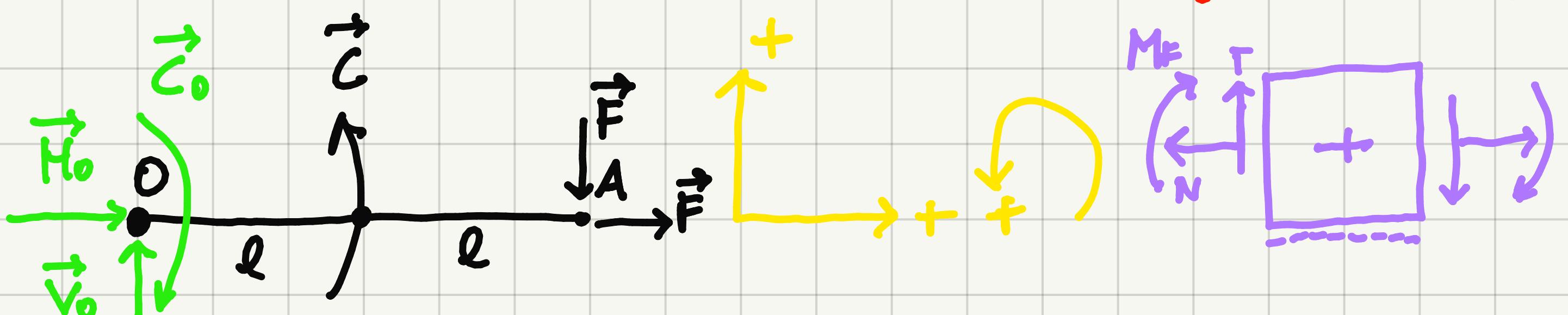
① Si consideri un'asta lineare incastriata ad una parete verticale in O



E LUNGA $\overline{OA} = 2l$. In A sono applicate una forza \vec{F} verso il basso ed una \vec{F} lungo l'orizzontale a destra. Esterna all'asta

ED UNA COPPIA ANTORATORIA APPUCIATA AL BANCHEMDO $|\vec{C}| = 3F\ell$.

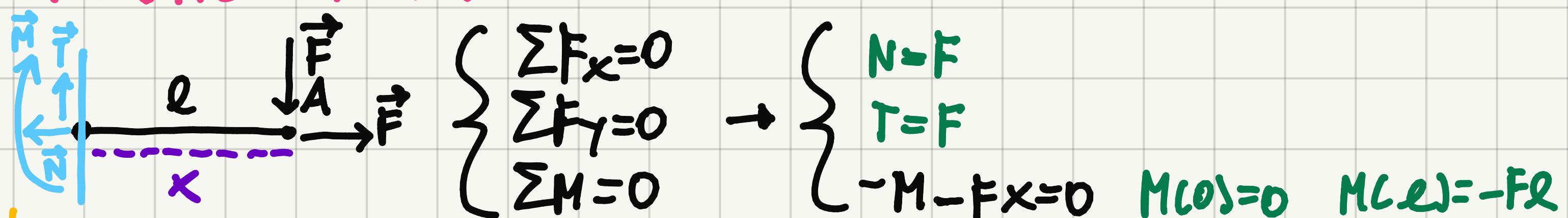
DETERMINARE LE REAZIONI VINCOLARI ED IL DIAGRAMMA DELLE AZIONI INTERNE



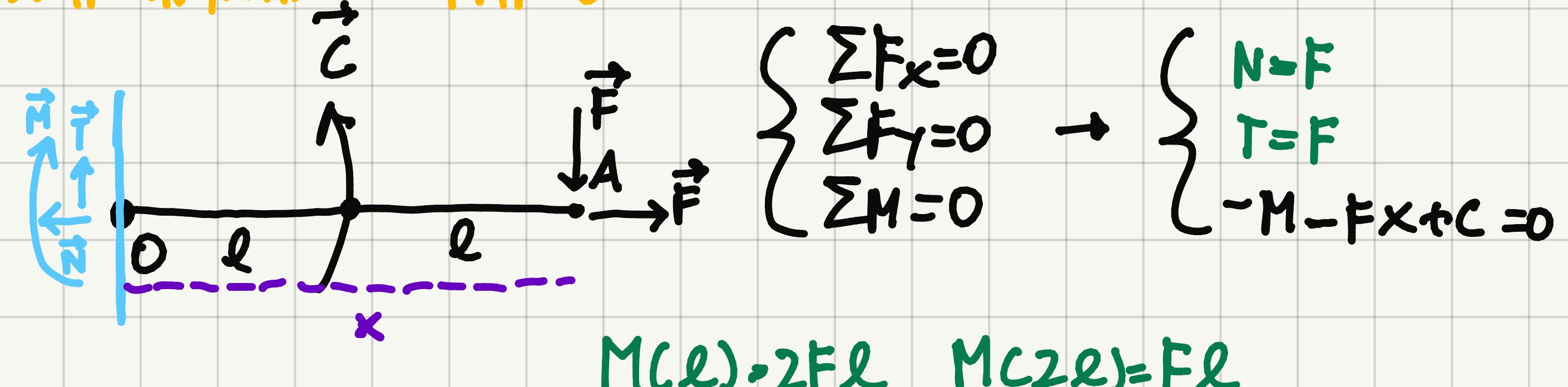
$$\begin{cases} \sum F_x = 0 \\ \sum F_y = 0 \\ \sum M_O = 0 \end{cases} \Rightarrow \begin{cases} H_0 + F = 0 \\ V_0 - F = 0 \\ -C_0 + C - F2L = 0 \end{cases}$$

$$\begin{aligned} H_0 &= -F \\ V_0 &= F \\ C_0 &= F\ell \end{aligned}$$

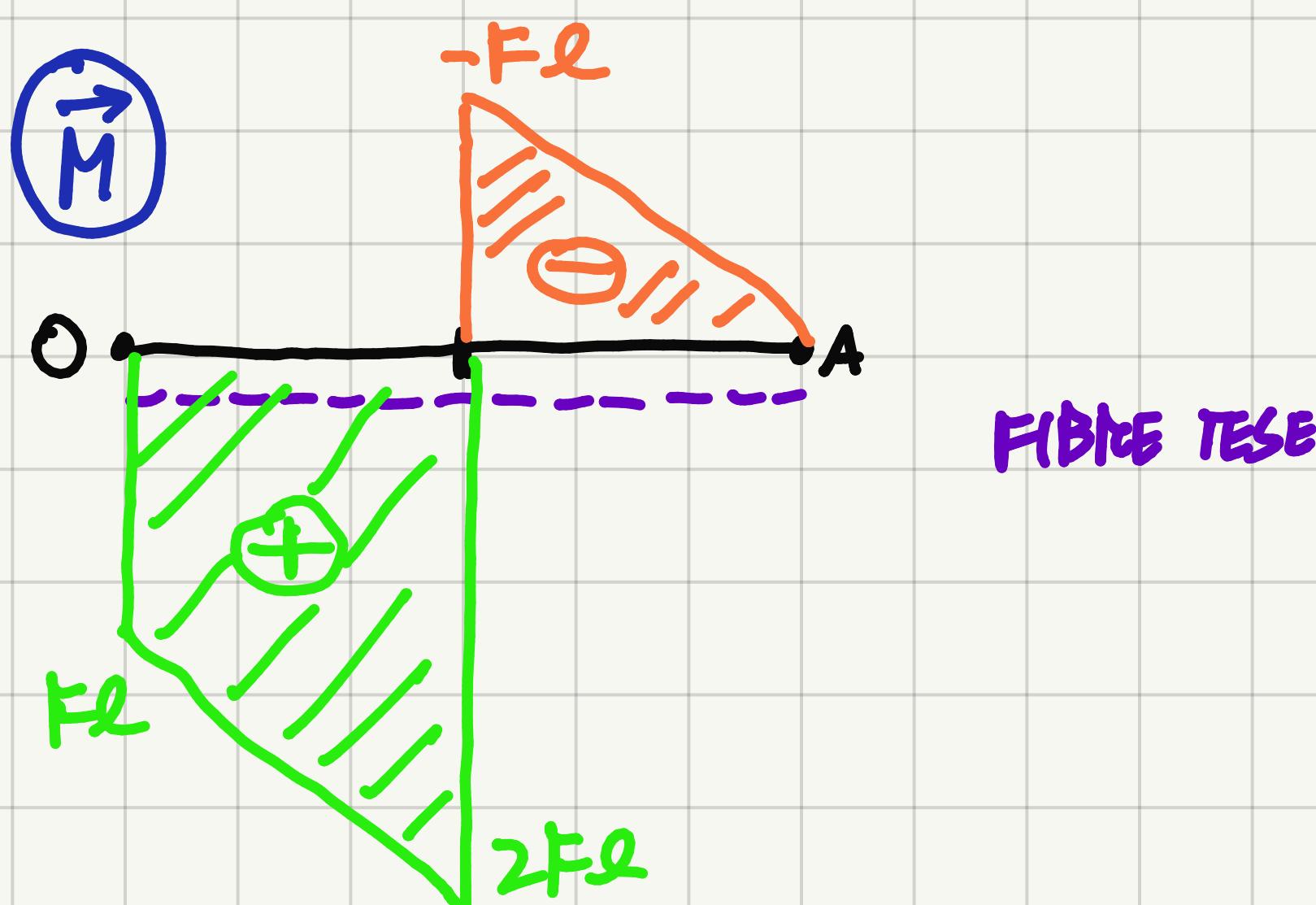
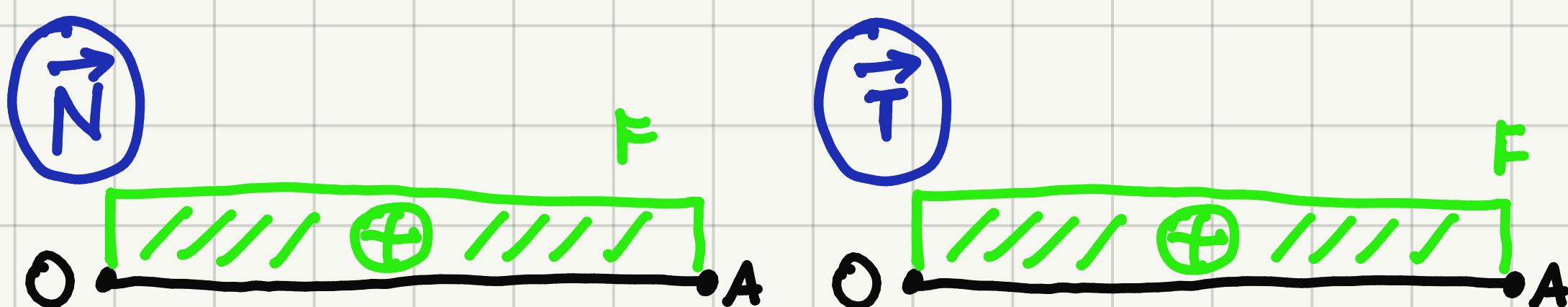
STUDIO AZIONI INTERNE



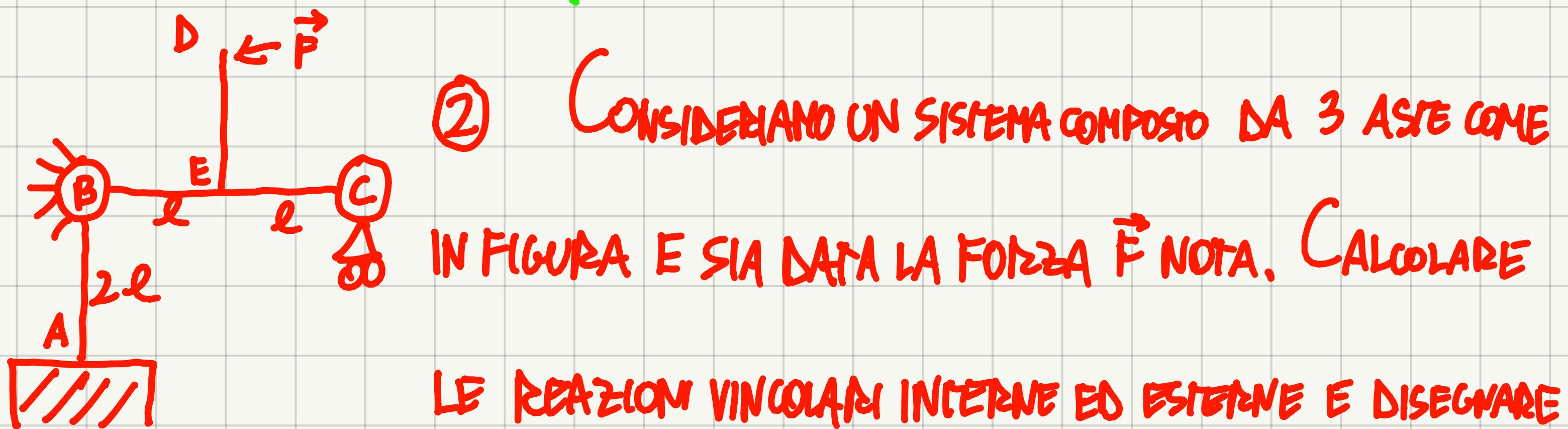
! SE IN UN TRATTO $T=0, M_F=0$



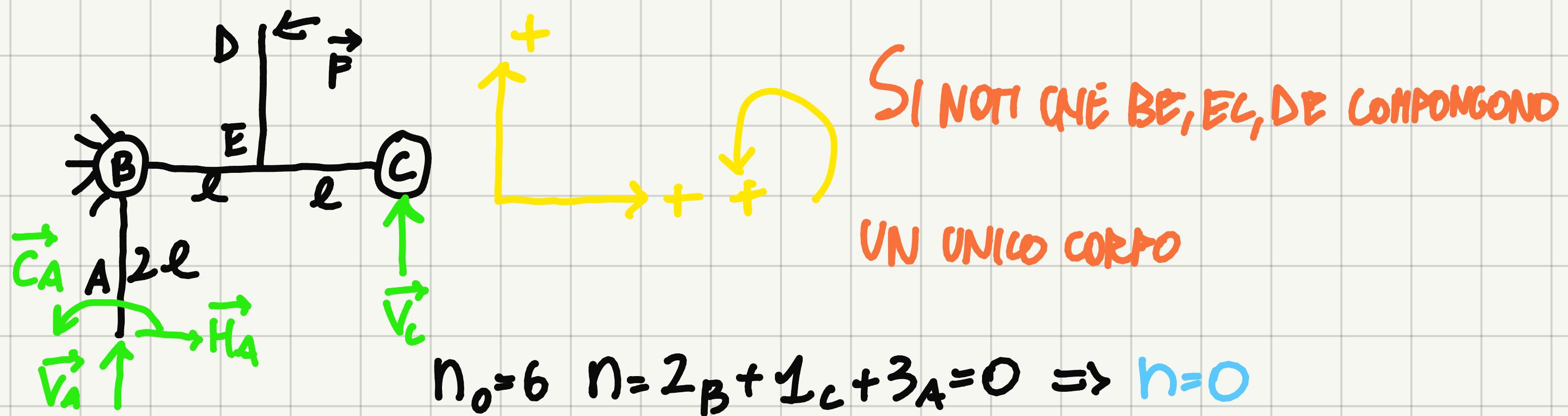
$$M(\ell) = 2Fl \quad M(2\ell) = Fl$$



② CONSIDERANDO UN SISTEMA COMPOSTO DA 3 ASTI COME

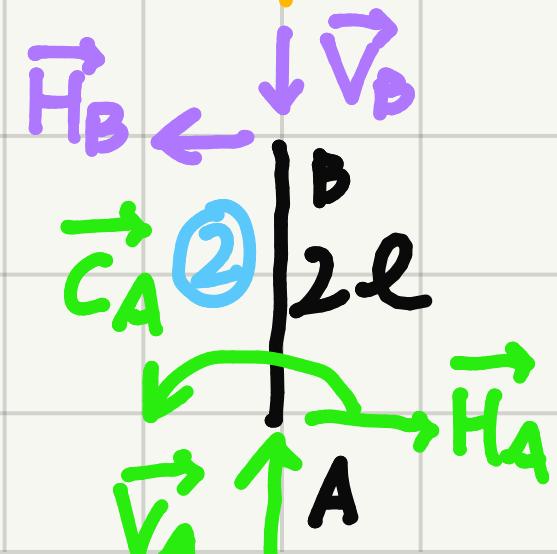


I DIAGRAMMI DELLE AZIONI INTERNE

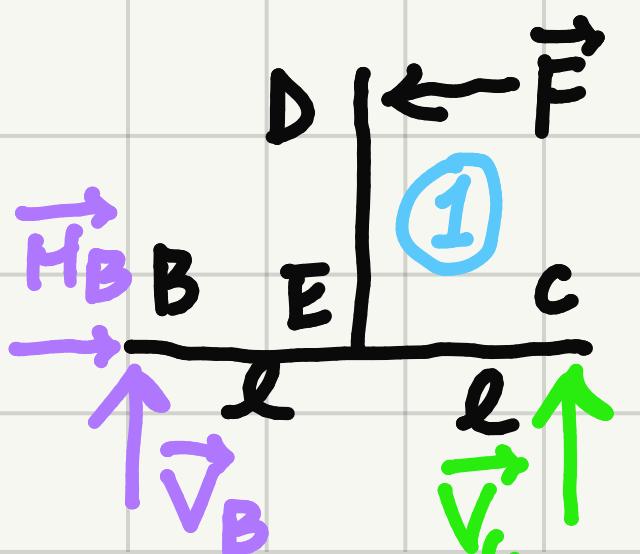


! LE 3 EQUAZIONI SONO INSUFFICIENTI PER CALCOLARE LE 4 INCognITE

⇒ SPEZZO I DUE CORPI



! LE FORZE IN B, NEI DUE CORPI, SONO UGUALI IN MODULO E OPPoste IN SEGNO



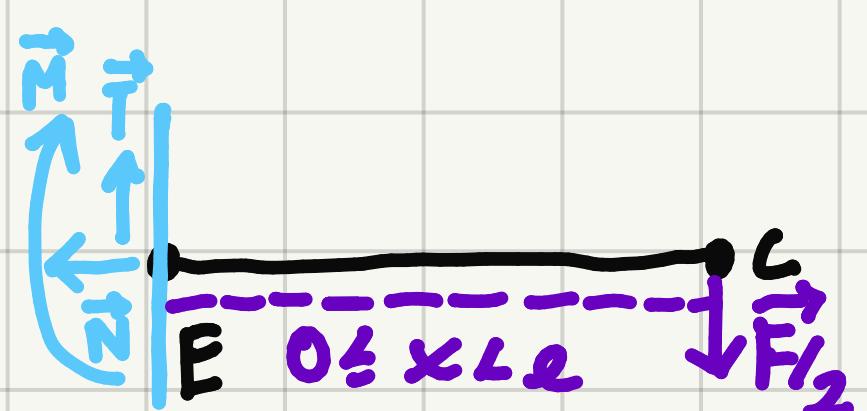
$$1 \begin{cases} H_B - F = 0 \\ V_B + V_C = 0 \\ F_l + 2lV_C = 0 \end{cases} \Rightarrow V_B = F/2 \quad V_C = -F/2$$

POSSIAMO ORA STUDIARE L'INTERO CORPO

$$\begin{cases} H_A - F = 0 \\ V_A + V_C = 0 \\ C_A + 2\ell H_A + 2\ell V_C + Fl = 0 \end{cases} \Rightarrow H_A = F, V_A = F/2, C_A = -2Fl$$

STUDIO AZIONI INTERNE

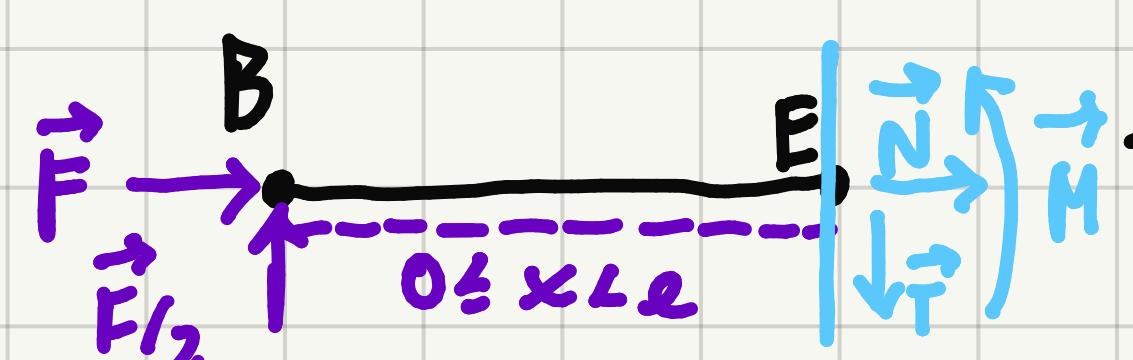
CE



$$\begin{cases} -N = 0 \\ T - F/2 = 0 \\ -M - F_{1/2}x = 0 \end{cases}$$

$$\Rightarrow \begin{cases} N = 0 & M(0) = 0 \\ T = F/2 & \\ M = F_{1/2}x & M(2l) = -F_{1/2}l \end{cases}$$

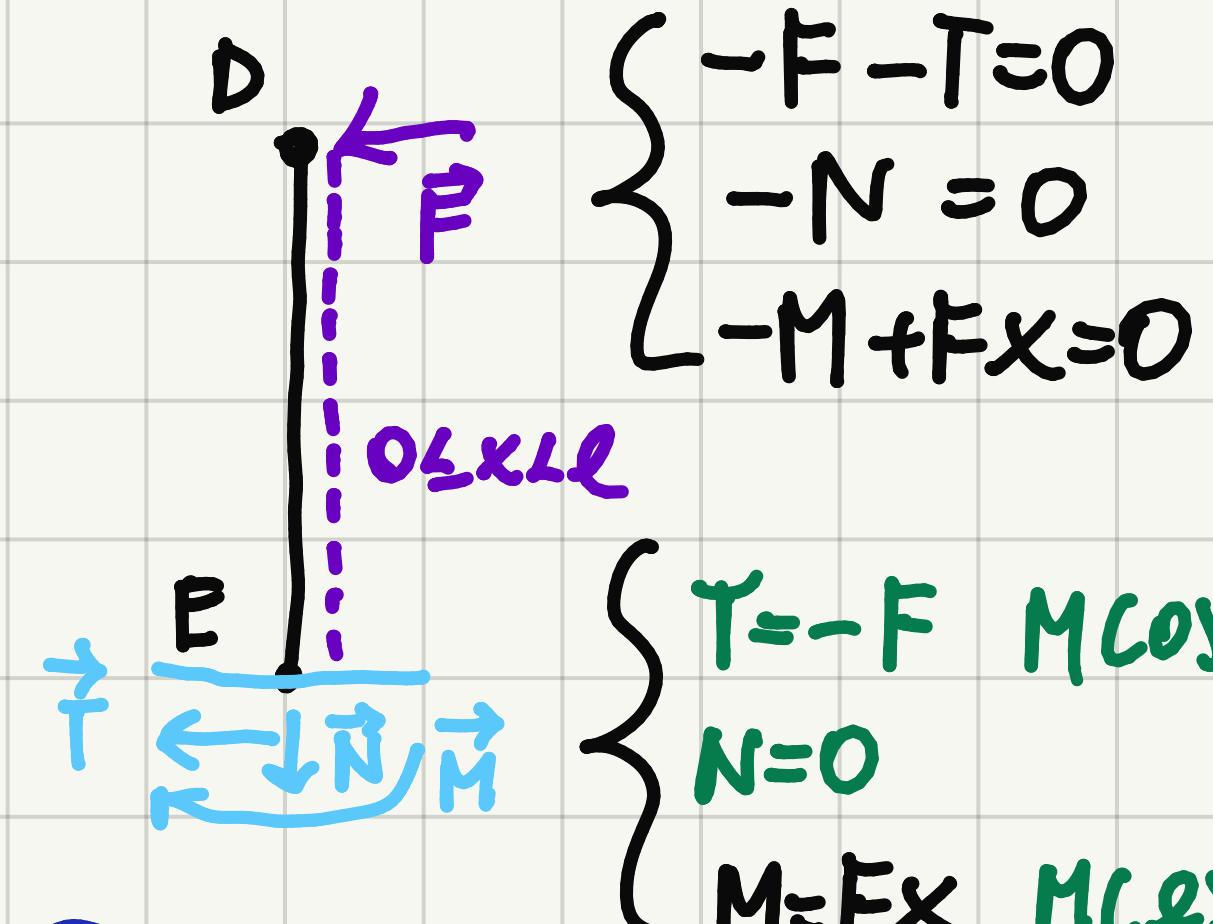
BE



$$\begin{cases} N + F = 0 \\ F_{1/2} - T = 0 \\ M - F_{1/2}x = 0 \end{cases}$$

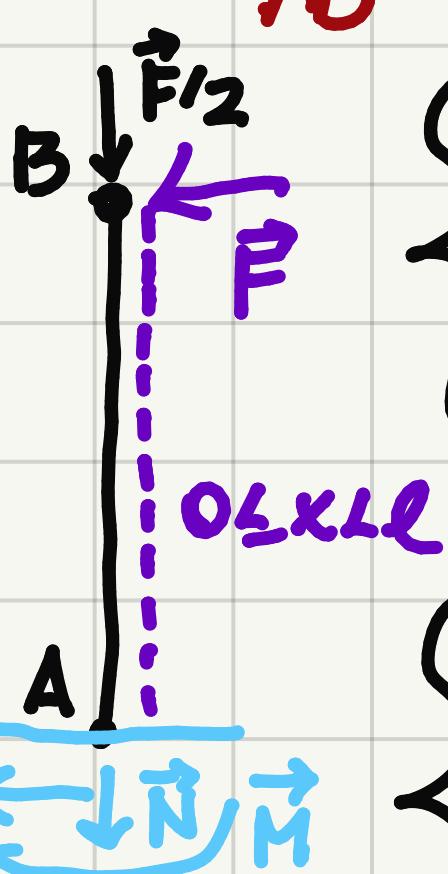
$$\Rightarrow \begin{cases} N = -F & M(0) = 0 \\ T = F/2 & \\ M = F_{1/2}x & M(2l) = F_{1/2}l \end{cases}$$

DE



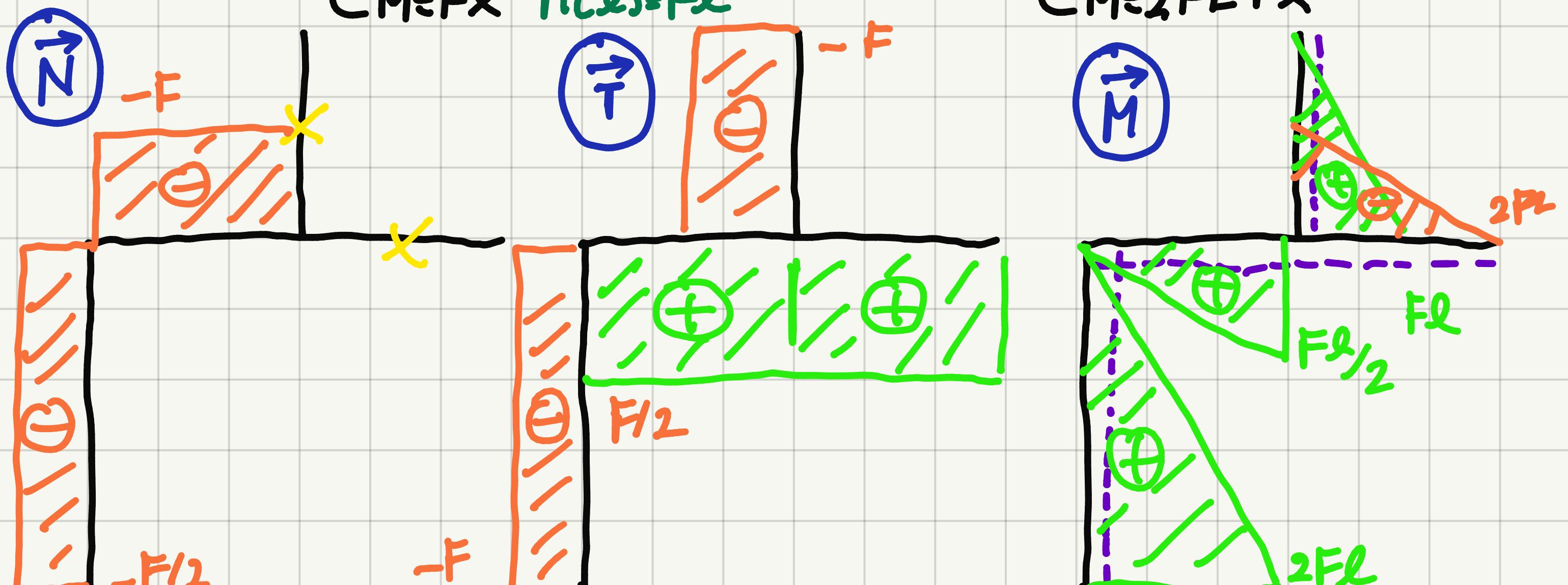
$$\begin{cases} -F - T = 0 \\ -N = 0 \\ -M + Fx = 0 \end{cases}$$

AB

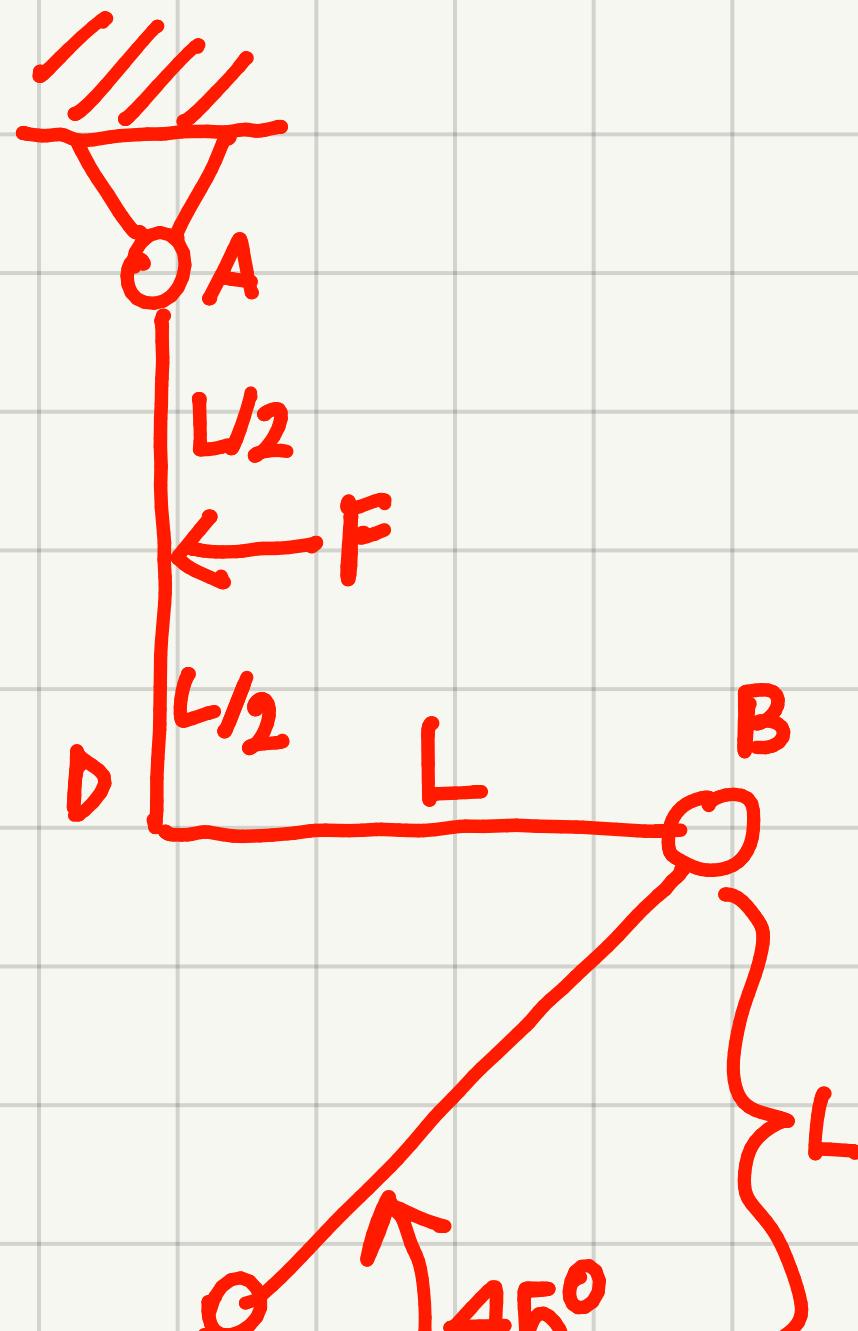


$$\begin{cases} -F - T = 0 \\ -N - F_{1/2} = 0 \\ -M + 2FL - Fx = 0 \end{cases}$$

$$\begin{matrix} M(0) = 2FL \\ M(2L) = 0 \end{matrix}$$



③

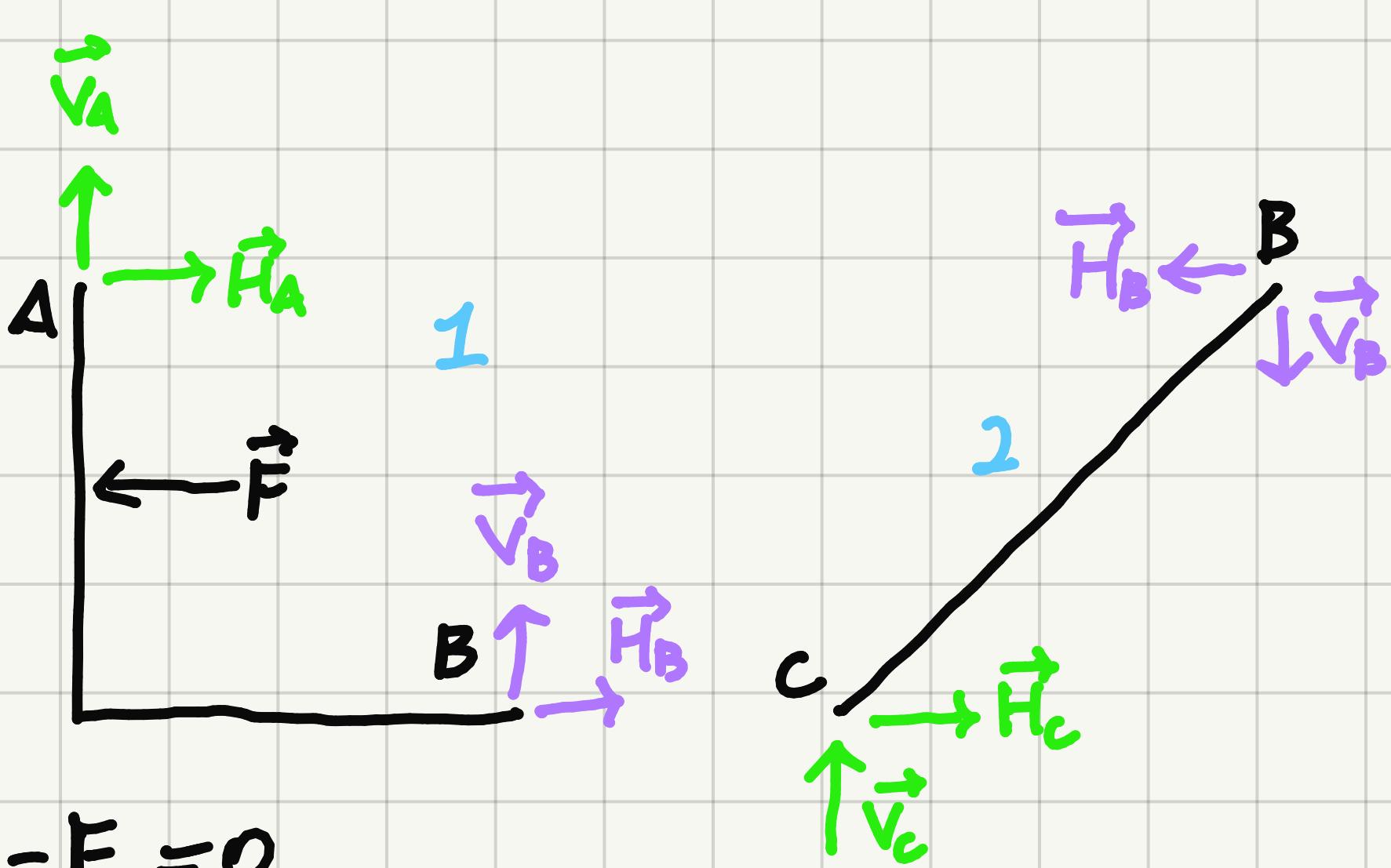


PER IL SISTEMA IN FIGURA, SI CALCOLINO
I GRADI DI LIBERTÀ, LE REAZIONI VINCOLARI
E DISSEGNARE IL DIAGRAMMA DELLE AZIOMI INTERNE

$$n_0 = 3 \cdot 2 = 6 \quad n_r = 2_A + 2_B + 2_C = 6$$

$$\Rightarrow n = 0$$

4 EQUAZIONI, 3 INCognITE \Rightarrow SPEZZO I DUE CORPI



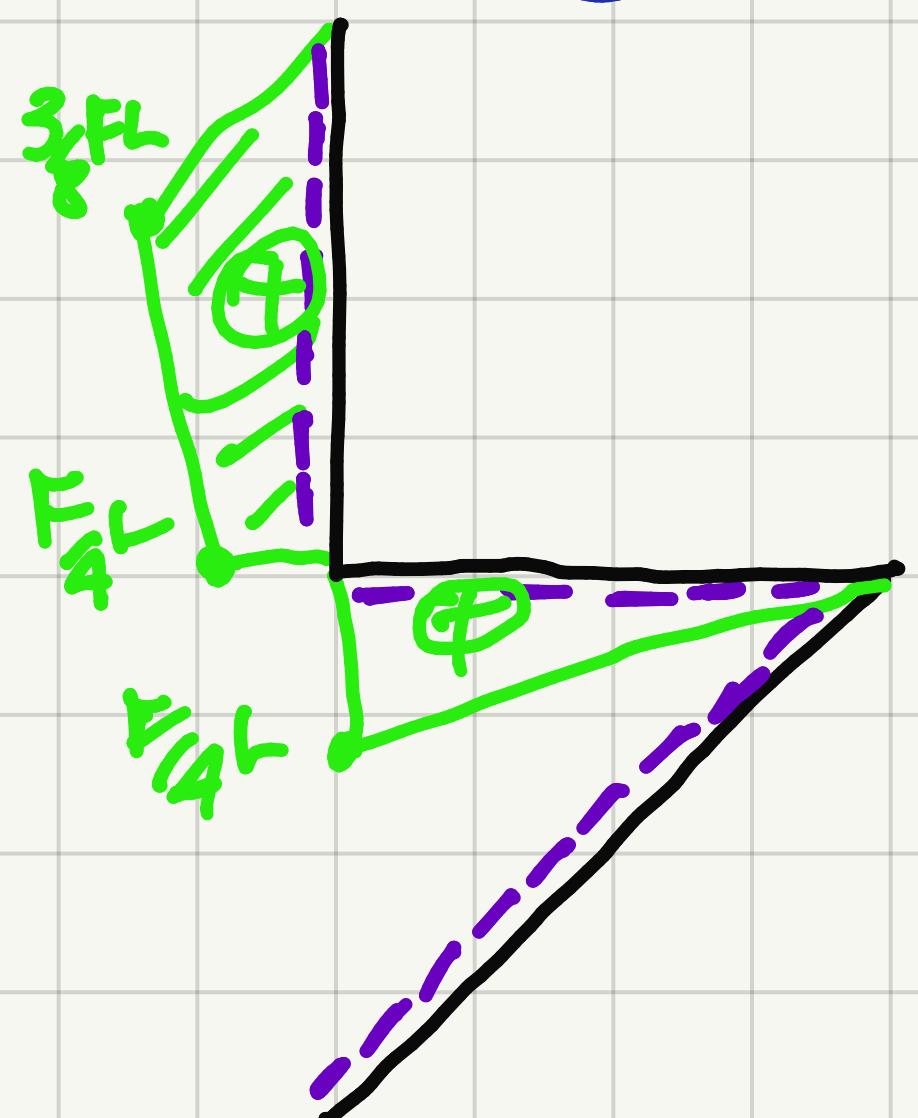
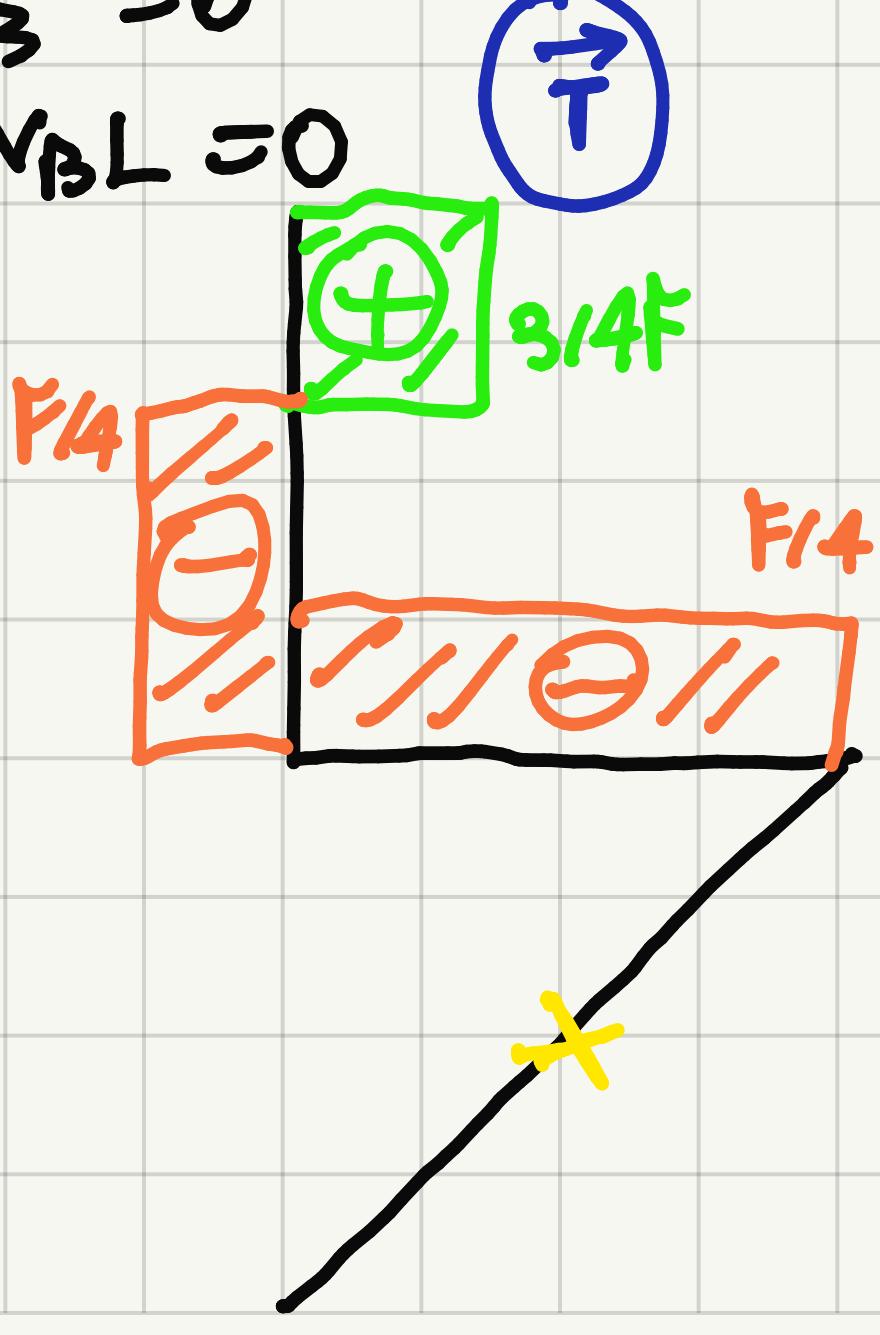
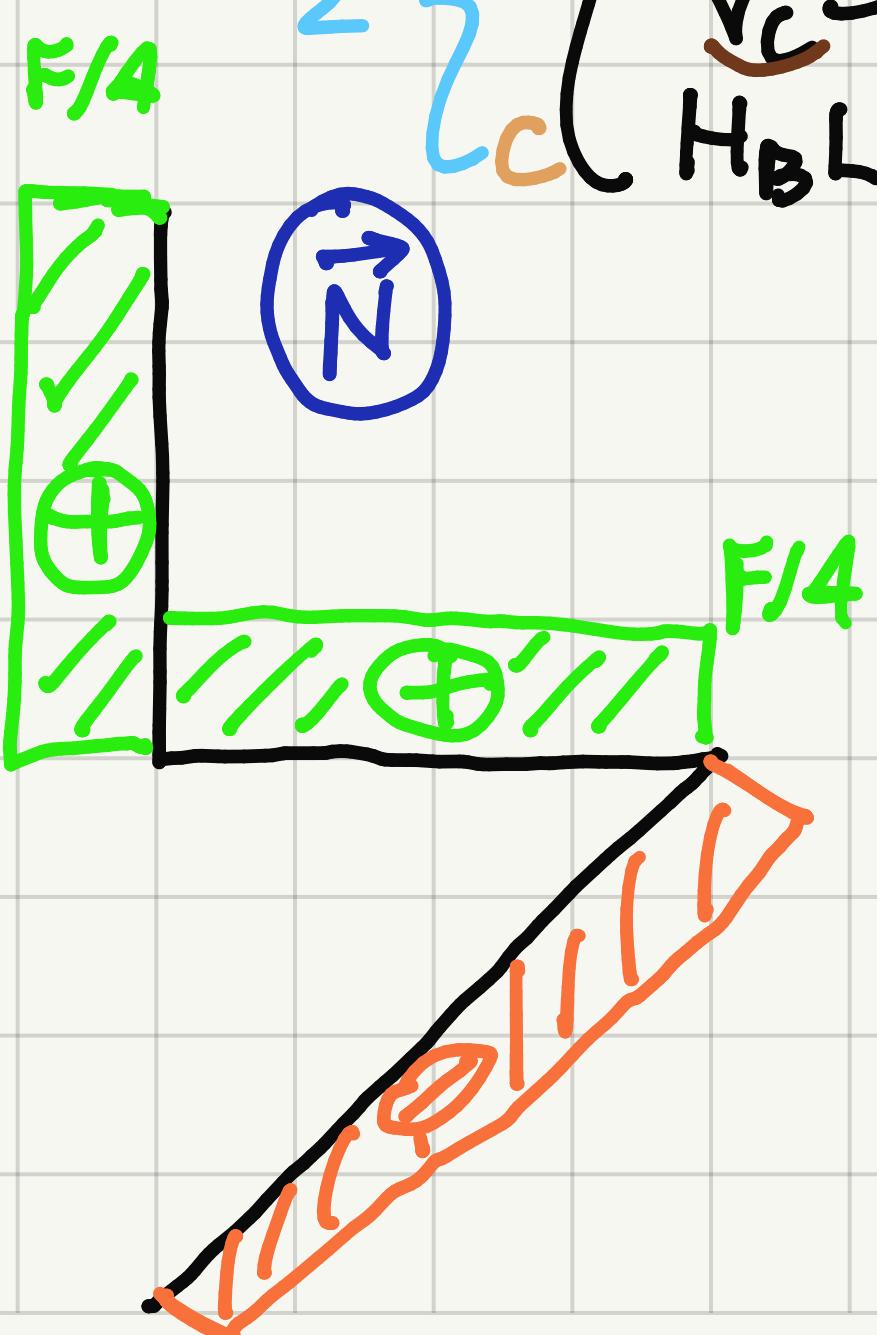
$$\left. \begin{array}{l} H_A + H_B - F = 0 \\ V_A + V_B = 0 \end{array} \right\}$$

$$F \cdot L/2 - H_A \cdot L - V_A \cdot L = 0$$

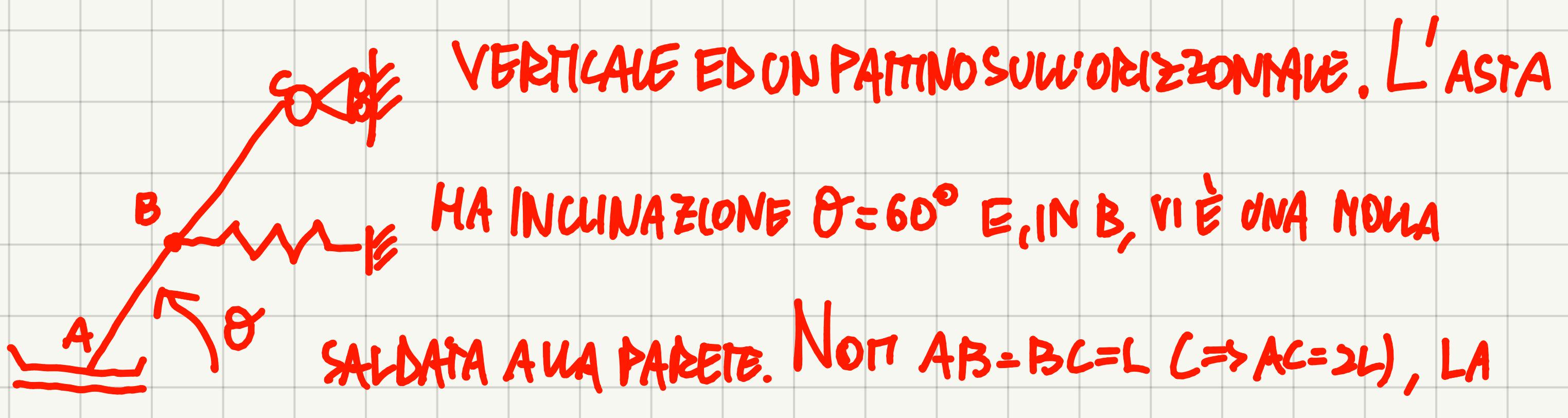
$$\left. \begin{array}{l} H_C - H_B = 0 \\ V_C - V_B = 0 \end{array} \right\}$$

$$H_B \cdot L - V_B \cdot L = 0$$

6 EQUAZIONI, 6 INCognITE



④ CONSIDERIAMO UN'ASTA AC VINCOLATA CON UN CARRELLO SULLA



COSTANTE DI ELASTICITÀ K E LA LUNGHEZZA INDEFORMATA DELLA MOLLA

$\ell_0 = 0$, CALCOLARE LE REAZIONI VINCOLARI E LE AZIONI INTERNE

$$\left\{ \begin{array}{l} \sum F_x = 0 \\ \sum F_y = 0 \\ \sum M_A = 0 \end{array} \right. \quad n_0 = 3 \quad n_d = 2_A + 1_C = 3 \Rightarrow n = 0$$

$$\left\{ \begin{array}{l} F_e - H_c = 0 \\ V_A = 0 \\ C_A - F_e L \sin \theta + H_c 2L \sin \theta = 0 \end{array} \right.$$

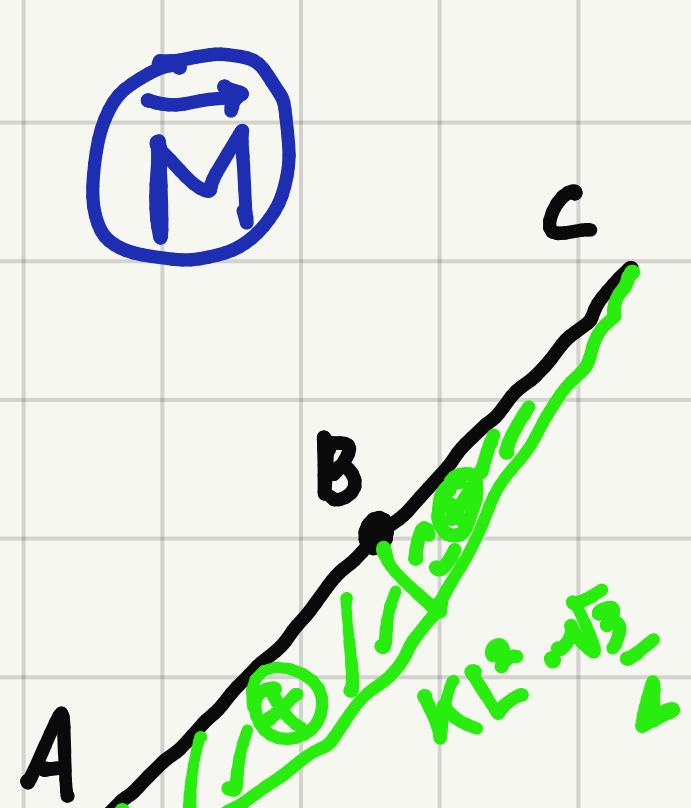
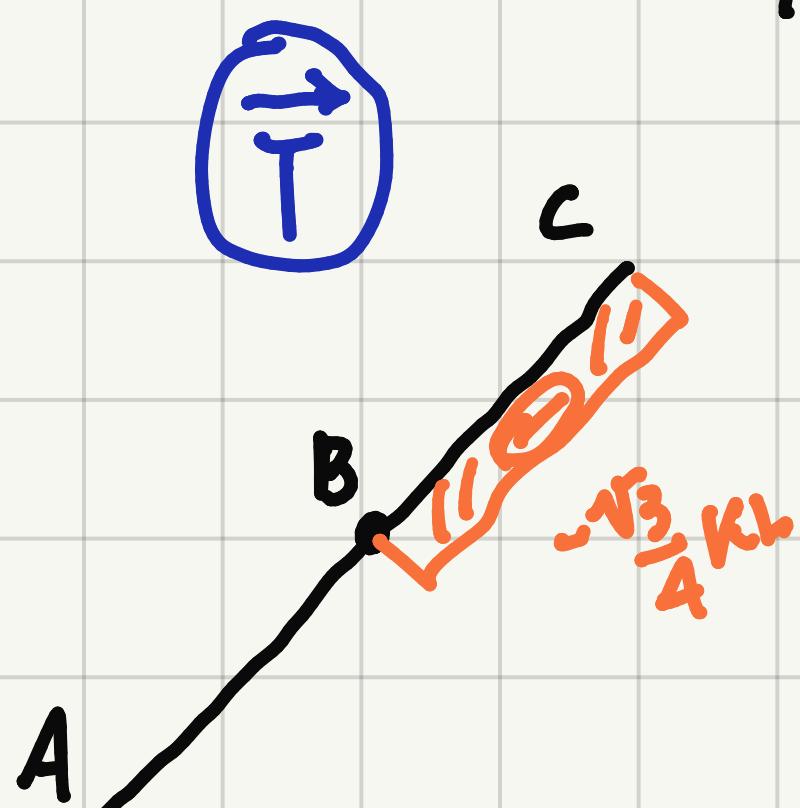
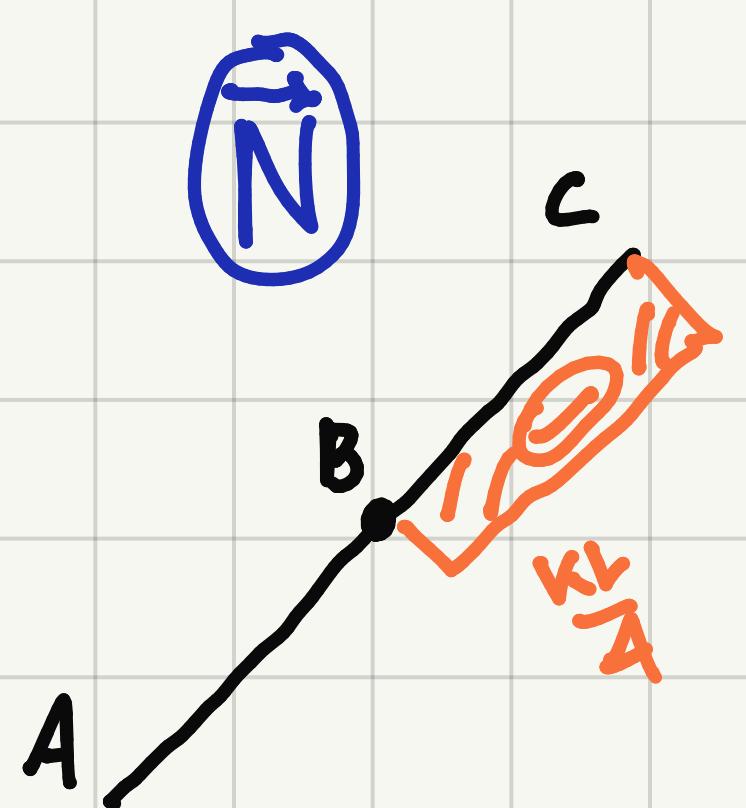
$$\left\{ \begin{array}{l} H_c = kL/2 \\ V_A = 0 \\ C_A = kL^2/2 \sin \theta - KL^2 \sin \theta = -KL^2/2 \end{array} \right. \quad F_e = k(L - \ell_0) = kL$$

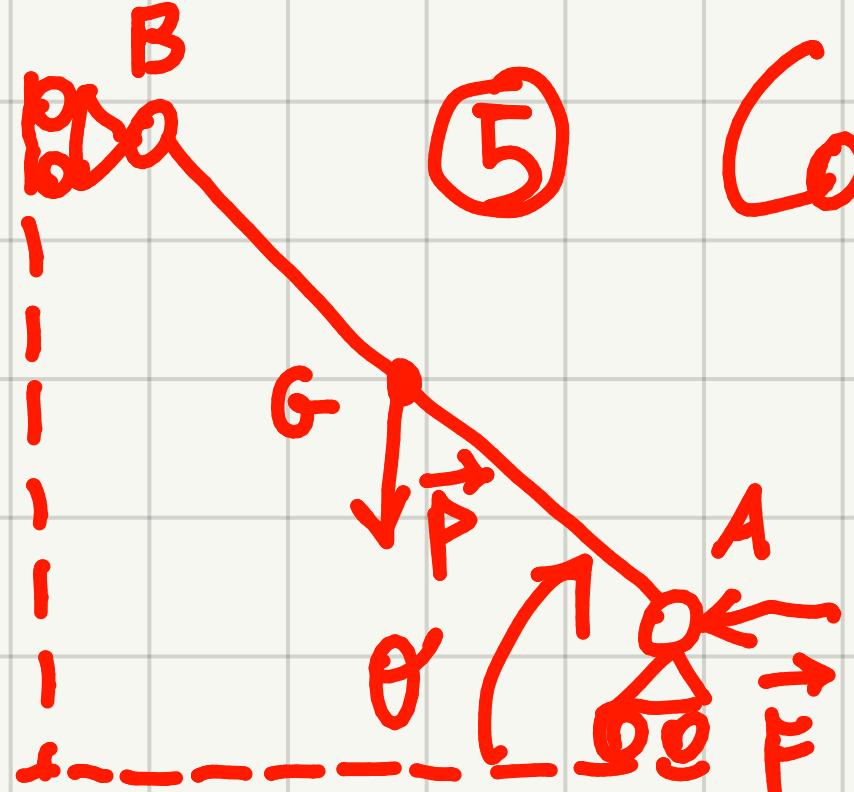
$$C_A = \frac{kL^2}{2} \sin \theta - KL^2 \sin \theta = -\frac{KL^2}{2} \sqrt{\frac{3}{2}} = -\frac{KL^2 \sqrt{3}}{4}$$

STUDIO AZIONI INTERNE

$$\left\{ \begin{array}{l} N = -\frac{KL}{2} \cos \theta = -\frac{KL}{4} \\ T = -\frac{KL}{2} \sin \theta = -\frac{\sqrt{3}}{4} KL \\ M(x) = \frac{KL}{2} \sin \theta \cdot x = \frac{\sqrt{3}}{4} KLx \end{array} \right. \quad M(0) = 0 \quad M(L) = \frac{\sqrt{3}}{4} KL^2$$

$$\left\{ \begin{array}{l} N = 0 \\ T = 0 \\ M_F = KL^2 \frac{\sqrt{3}}{4} \end{array} \right.$$





5 CONSIDERARE UNA SCALA CHE SIOSSE SUL PIANO VERTICALE.
AL CENTRO DELLA STA VIÈ IL BARILENTO IN CUI È
APPICATA LA FORZA PESO. NOTA LA LUNGHEZZA

DELL'ASTA $\overline{AB} = l$, LA MASSA m , L'INERZIA I DEL CORPO E L'ANGOLI DI
EQUILIBRIO θ , DETERMINARE IL VALORE DELLA FORZA \vec{F} CHE MANTENGA

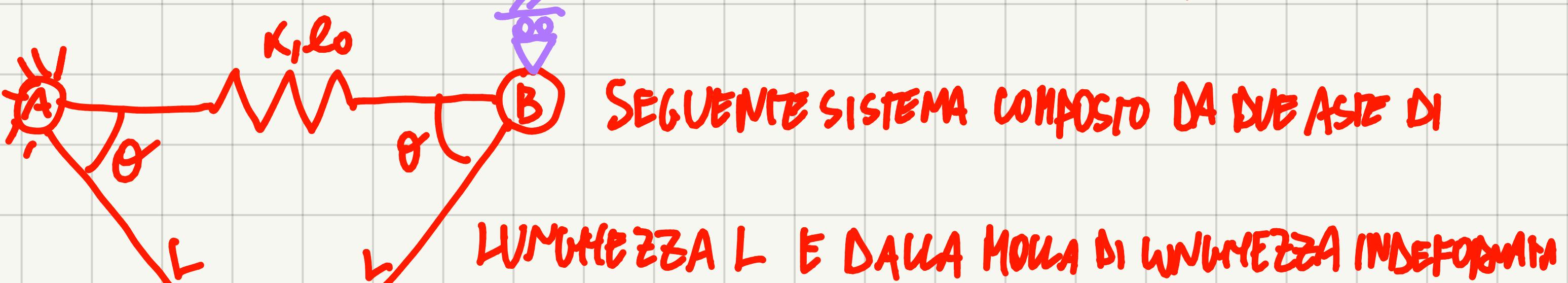
IL SISTEMA IN EQUILIBRIO STATICO

$$n_0 = 3 \quad n_v = 1_B + 1_A + 1_F = 3 \quad \Rightarrow \quad n = 0$$

$$\begin{aligned} \sum \vec{F}_x &= 0 \\ \sum \vec{F}_y &= 0 \\ \sum \vec{M}_A &= 0 \end{aligned} \quad \left\{ \begin{array}{l} H_B - F = 0 \rightarrow F = H_B \\ V_A - P = 0 \rightarrow V_A = P \\ P \frac{l}{2} \cos \theta - H_B \frac{l}{2} \sin \theta = 0 \end{array} \right.$$

$$F = P_{1/2} / \cos \theta$$

6 SI CALCOLINO I GRADI DI LIBERTÀ E LE RIBAZIONI VINCOLARI DEL



$$n_0 = 6 \quad n_v = 2_A + 1_B + 2_C = 6 \Rightarrow n = 1 \quad \text{ACB MANOVEMENTO ORDINARIO}$$

DELE AZIONI INTERNE
 $F_e = K(l - l_0) = K(2l \cos \theta - \frac{L}{2})$ DI LIBERTÀ

$$\left\{ \begin{array}{l} H_A + F = 0 \\ V_A + V_B = 0 \\ FL \sin \theta + 2V_B L \cos \theta = 0 \end{array} \right.$$

$$H_A = -F$$

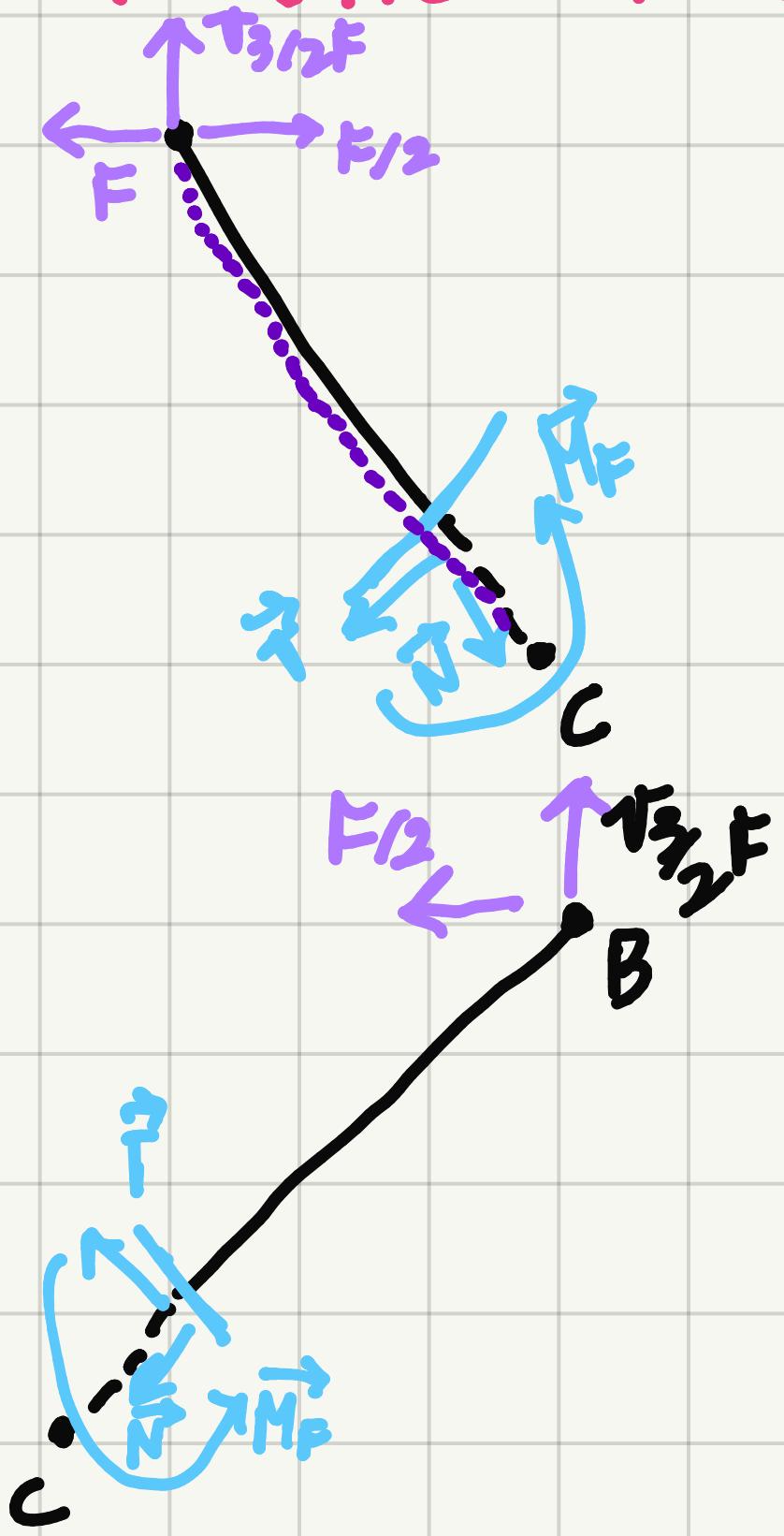
$$V_B = -F_{1/2} \tan \theta$$

$$V_A = -V_B$$

! \vec{F} VA MESSA SOLO IN UNA DUE DUE ASTE. SCELTA ARBITRARIA

$\left. \begin{array}{l} -M_C + F - F_e = 0 \\ V_B - V_C = 0 \\ L F_e \sin \theta + L V_B \cos \theta = 0 \end{array} \right\}$
 $\kappa (2L \cos \theta - L_{x_2}) \sin \theta - \frac{F}{2} \cos \theta \sin \theta = 0$
 $\theta = \arccos \left(\frac{-KL + 2kL_{x_2}}{4KL} \right) = 60^\circ \Rightarrow V_B = -\frac{\sqrt{3}}{2} F, V_A = \frac{\sqrt{3}}{2} F, F_e = F/2$

STUDIO AZIONI INTERNE



$$\left\{ \begin{array}{l} N + (F_{x_2} - F) \cos(\theta) + \frac{\sqrt{3}}{2} F \sin \theta = 0 \Rightarrow N = F \\ T = 0 \\ M_F - T x = 0 \quad \text{perché } T = 0, M_F = 0 \end{array} \right.$$

$$\left\{ \begin{array}{l} N + F_{x_2} \cos \theta + \frac{\sqrt{3}}{2} F \sin \theta = 0 \rightarrow N = -F \\ T + F_{x_2} \sin \theta - \frac{\sqrt{3}}{2} F \cos \theta = 0 \rightarrow T = 0 \\ -M_F + T x = 0 \Rightarrow M_F = 0 \end{array} \right.$$

