Récapitulatif de résistance des matériaux

P9-12 - Résumé

Sollicitation	$\{T_{coh}\}$	$\vec{C}(M,\vec{n})$	Formules / Remarques
Extension (N > 0) Compression (N < 0)	$ \begin{pmatrix} N & 0 \\ 0 & 0 \\ 0 & 0 \end{pmatrix}_{G} $	\sigmaec{n}	$N = \int_{S} \sigma dS$ $\sigma = \frac{N}{S}$ si effort uniforme $\sigma_{x} = E \mathcal{E}_{x}$ $\mathcal{E}_{x} = \frac{du(x)}{dx}$ $\mathcal{E}_{y} = \frac{dv(y)}{dy} = -v \mathcal{E}_{x}$
Cisaillement	$ \begin{cases} 0 & 0 \\ T_y & 0 \\ T_z & 0 \end{cases}_G $	$ auec{t}$	$T = \int_{S} \tau dS$ $\tau = \frac{T}{S}$ si τ uniforme $\tau = G\gamma$ $\tau = G\gamma$ $\tau = \frac{dy}{dx}$ $\sigma = \frac{E}{2(1+\nu)}$
Torsion simple	$egin{cases} egin{pmatrix} 0 & \overrightarrow{m_t} \ 0 & 0 \ 0 & 0 \end{pmatrix}_G$	$\tau \overrightarrow{z'}$	$\boxed{\theta = \frac{d\alpha}{dx}} \qquad \boxed{\tau = G\theta\rho \atop \rho = GM} \qquad \boxed{m_t = G\theta I_0} \qquad \tau = \frac{m_t}{I_0}\rho$
Flexion plane simple	$egin{cases} egin{pmatrix} 0 & 0 \ T_y & 0 \ 0 & m_{f_z} \end{pmatrix}_G$	$\sigma \vec{n} + \tau \vec{t}$	
(appuis)	$ \begin{cases} -F & 0 \\ 0 & 0 \\ 0 & -yF \end{cases}_{G} $	$y'' + \frac{F}{EI_{GZ}}y$	
Flambement (encastré)		$y'' + \frac{F}{EI_{GZ}}y$	$y = \frac{m_B}{E I_{GZ}} \boxed{y = -\frac{m_B}{F} \cos\left(\frac{2k\pi}{l}x\right)} y_G = A\cos(\omega x) + B\sin(\omega x) y_P = \frac{m_B}{F} \begin{cases} \text{Appui simple} \; / \; Appui simple} \; / \; L = l \\ \text{Appui simple} \; / \; Encastrement} \; / \; Encastr$

 (σ_e, τ_e) Limite élastique :

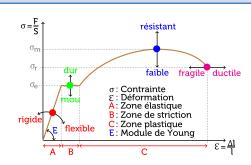
 (σ_r, τ_r) Limite à la rupture :

Critère de Rankine : $\sigma_{max} \leq \sigma_e$

Critère de Guest : $\tau_{max} \le \tau_e$

Coef. de sécurité :

Coef. de concentr° de contrainte : $\sigma_{max} = a \ \sigma_{rdm}$



Valeurs courantes:

 $\nu_{\rm acier} = 0.3$

 $E_{\text{fontes}} = 60\text{-}160 \, MPa$

 $E_{\text{acier}} = 210 \text{ MPa}$ $E_{\text{cuivre}} = 120 \text{ MPa}$ $E_{\text{alu}} = 70 \text{ MPa}$

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I. **Torseur**

1. Définition

$$\{T\} = \left\{ \overrightarrow{R} \atop \overrightarrow{m_A} \right\}_A = \left\{ \overrightarrow{R} \atop \overrightarrow{m_B} = \overrightarrow{m_A} + \overrightarrow{BA} \wedge \overrightarrow{R} \right\}_B = \left\{ \begin{matrix} R_x & m_x \\ R_y & m_y \\ R_z & m_z \end{matrix} \right\}_A$$
• $\overrightarrow{m_P} \cdot \overrightarrow{PM} = \overrightarrow{m_M} \cdot \overrightarrow{PM}$
Equiprojectivité
• $C = \overrightarrow{R} \cdot \overrightarrow{m_P} = \overrightarrow{R} \cdot \overrightarrow{m_M}$
Invariant scalaire

Classification

$$(C = 0)$$
:

$$\{\mathcal{T}\} = \begin{Bmatrix} 0 \\ 0 \end{Bmatrix}$$

$$cst \ \forall \ M$$

Torseur glisseur

Torseur général

$$(C = 0)$$
:

$$\{\mathcal{T}\} = \left\{\begin{matrix} 0 \\ \overrightarrow{m} \end{matrix}\right\}_{M}$$

$$(C = 0)$$
:

$$(C = \mathbf{0}): \qquad \{\mathcal{T}\} = \begin{Bmatrix} 0 \\ 0 \end{Bmatrix}_{M} \qquad cst \ \forall M$$

$$(C = \mathbf{0}): \qquad \{\mathcal{T}\} = \begin{Bmatrix} 0 \\ \overrightarrow{m} \end{Bmatrix}_{M} \qquad cst \ \forall M$$

$$(C = \mathbf{0}): \qquad \{\mathcal{T}\} = \begin{Bmatrix} \overrightarrow{R} \\ \overrightarrow{m_{A}} \end{Bmatrix}_{A} \qquad \overrightarrow{m_{A}} \perp \overrightarrow{R}$$

$$(C \neq \mathbf{0}): \qquad \{\mathcal{T}\} = \left\{\begin{matrix} \vec{R} \\ \vec{m_A} \end{matrix}\right\}_A = \left\{\begin{matrix} \vec{R} \\ \vec{m_A} \end{matrix}\right\}_A + \left\{\begin{matrix} 0 \\ \vec{m_A^2} \end{matrix}\right\}_A + \left\{\begin{matrix} 0 \\ \vec{m_A^2} \end{matrix}\right\}_A$$
Torseur gibsseur Torseur coup

$$\overrightarrow{m_A} \perp$$

Torseur coupl
$$\overrightarrow{m_A^2} \parallel \vec{R}$$

2. Propriétés

 $\bullet \quad \mathcal{P} = \overrightarrow{R_1} \cdot \overrightarrow{m_{2_P}} = \overrightarrow{R_2} \cdot \overrightarrow{m_{1_P}}$ Comoment

Axe du glisseur : $\overrightarrow{AP} = \frac{\overrightarrow{R} \wedge \overrightarrow{m_A}}{R^2} + \lambda \overrightarrow{R}$

 $\lambda = 0 \Leftrightarrow \vec{m} = 0$

Principe fondamental de la statique (PFS) II.

$$\boxed{\left\{\mathcal{T}_{\bar{e}/e}\right\} = \left\{0\right\}} \quad \Rightarrow \quad \begin{cases} \overrightarrow{R_{\bar{e}/e}} = \overrightarrow{0} \\ \overrightarrow{m_{\bar{e}/e}} = \overrightarrow{0} \end{cases} \quad (\exists \ M \Rightarrow \forall \ F)$$

III. Torseur de cohésion

$$\begin{cases} \{T_{coh}\}_G = \{T_{E_2/E_1}\} = -\{T_{\bar{E}/E_1}\} = \{T_{\bar{E}/E_2}\} = \begin{cases} N & m_t \\ T_y & m_{f_y} \\ T_z & m_{f_z} \end{cases}_G \end{cases}$$

$$\begin{cases} \frac{dN}{dx} = -f(x) & \frac{dm_t}{dx} = 0 \\ \frac{dT_y}{dx} = -p(x) & \frac{dm_{f_y}}{dx} = T_z(x) \\ \frac{dT_z}{dx} = -q(x) & \frac{dm_{f_z}}{dx} = -T_y(x) \end{cases}$$

$$\frac{dN}{dx} = -f(x) \qquad \frac{dm_t}{dx} = 0$$

$$\frac{dT_y}{dx} = -p(x) \qquad \frac{dm_{f_y}}{dx} = T_z(x)$$

$$\frac{dT_z}{dx} = -q(x) \qquad \frac{dm_{f_z}}{dx} = -T_y(x)$$

IV. Vecteur contrainte

$$\boxed{\vec{C}(M,\vec{n}) = \frac{d\vec{f}}{dS} = \sigma \,\vec{n} + \tau \,\vec{t} \qquad \{\mathcal{T}_{coh}\} = \begin{cases} \int_{S} \vec{C}(M,\vec{n}) \,dS \\ \int_{S} \vec{GM} \wedge \vec{C}(M,\vec{n}) \,dS \end{cases}}$$

Moments quadratiques

$$I_{G} = \int_{S} \rho^{2} dS$$
moment quadratique
polaire (mm⁴)

$$\begin{bmatrix} I_{Gy} - \int_{S}^{Z} us \\ \text{moment quadratique} \\ \text{autour de l'axe } (G, \vec{y}) \end{bmatrix}$$

$$I_{GZ} = \int_{S} y^2 \, dS$$

autour de l'axe (G,\vec{z})

$$I_{Gyz} = \int_{S} yz \, dS$$
moment quadratique

$$I_G = I_{Gy} + I_{Gz}$$

1. Théorème de Huygens

$$I_{0} = (y_{G}^{2} + z_{G}^{2})S + I_{G}$$

$$I_{0y} = z_{G}^{2}S + I_{Gy}$$

$$I_{0z} = y_{G}^{2}S + I_{Gz}$$

2. Valeurs importantes

Circulaire	$I_G = \frac{\pi D^4}{32}$	$I_{Gy} = I_{Gz} = \frac{I_G}{2}$
Circulaire creuse	$I_G = \frac{\pi (D^4 - d^4)}{32}$	$I_{Gy} = I_{Gz} = \frac{I_G}{2}$
Rectangulaire	$I_G = \frac{hb(b^2 + h^2)}{12}$	$I_{Gy} = \frac{h^3 b}{12} \qquad I_{Gz} = \frac{hb^3}{12}$
Carré	$I_G = \frac{a^4}{6}$	$I_{Gy} = I_{Gz} = \frac{I_G}{2}$