9.2.4 The Rate of Heat Conduction

The rate of heat flow in solids depends on:

- ➤ Geometry of the solid (plate, cylinder, sphere)
- > Temperature distribution
- \triangleright Thermal conductivity, κ

We will look at the simpler case of steady-state heat flow only. Steady-state means the temperature distribution is not changing with time. The simplest geometry is that of a flat plate in which the cross-sectional area for heat flow remains constant.

$$Q = -\mathbf{k}A \frac{dT}{dx} = -\mathbf{k}A \frac{\Delta T}{\Delta x}$$

If you know κ , A and the temperature gradient, the amount of heat energy flowing through the surface can be calculated.

Composite (Layered) Walls

- ➤ Good heat insulators often do not have good mechanical properties.
- Composite walls are used where good heat insulation needs to be combined with good mechanical properties.
- ➤ Often several layers may be involved.

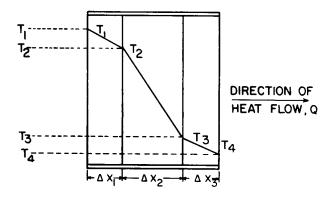


Figure 9-6 Temperature Profile for a Composite Wall

Analysis of heat flow through a composite wall

> At steady-state the amount of heat flow is same in every layer, while thermal conductivity and temperature gradients are different, i.e.

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$$Q_1 = Q_2 = Q_3 = Q$$

$$Q_1 = -\mathbf{k}_1 A \frac{T_2 - T_1}{\Delta x_1} = \mathbf{k}_1 A \frac{T_1 - T_2}{\Delta x_1}$$
 For layer # 1. Similarly for layer #2,

$$Q_2 = -\mathbf{k}_2 A \frac{T_3 - T_2}{\Delta x_2} = \mathbf{k}_2 A \frac{T_2 - T_3}{\Delta x_2}$$
 and for layer #3,

$$Q_3 = -\mathbf{k}_3 A \frac{T_4 - T_3}{\Delta x_3} = \mathbf{k}_3 A \frac{T_3 - T_4}{\Delta x_3}$$

These equations can be rearranged to make them explicit for the temperature difference. Also, Q_1 , Q_2 and Q_3 can be replaced by Q.

$$T_1 - T_2 = \frac{Q(\Delta x_1)}{\mathbf{k}_1 A}$$
 For layer #1

$$T_2 - T_3 = \frac{Q(\Delta x_2)}{\mathbf{k}_2 A}$$
 For layer #2

$$T_3 - T_4 = \frac{Q(\Delta x_3)}{\mathbf{k}_3 A}$$
 For layer #3

Taking the sum of these equations,

$$T_1 - T_4 = \frac{Q}{A} \left[\frac{\Delta x_1}{\mathbf{k}_1} + \frac{\Delta x_2}{\mathbf{k}_2} + \frac{\Delta x_3}{\mathbf{k}_3} \right]$$

This can be rearranged to get an expression for Q.

$$Q = \frac{A(T_1 - T_4)}{\left[\frac{\Delta x_1}{\mathbf{k}_1} + \frac{\Delta x_2}{\mathbf{k}_2} + \frac{\Delta x_3}{\mathbf{k}_3}\right]}$$

Resistance to heat flow (insulation value) is defined as $R = \frac{\Delta x}{k}$.

For the three layers, we can write

$$R_1 = \frac{\Delta x_1}{\boldsymbol{k}_1}$$

$$R_2 = \frac{\Delta x_2}{\mathbf{k}_2}$$

$$R_3 = \frac{\Delta x_3}{\mathbf{k}_3}$$

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The heat flow in terms of the resistances of the layers is

$$Q = \frac{(T_1 - T_4)A}{R_1 + R_2 + R_3} = \frac{A(T_1 - T_4)}{R}$$

$$or, \ \frac{Q}{A} = -\frac{\Delta T}{R}$$

Heat flux is proportional to ΔT and inversely proportional to R. Analogous to electric current (I = $\Delta E/R$)

Heat flow through cylindrical elements (pipes)

In this case, heat flows in the radial direction and is given by

$$Q = -\mathbf{k}A\frac{dT}{dr}$$

The area for heat flow for a given length L of the pipe is, $A = 2\pi rL$. Note that the area is not constant, it increases with r.

$$Q = -2\mathbf{p}\mathbf{k}rL\frac{dT}{dr}$$
or,
$$Q\frac{dr}{r} = -2\mathbf{p}\mathbf{k}LdT$$

Integrating both sides between r₁ and r₂ gives,

$$Q\int_{r_1}^{r_2} \frac{dr}{r} = -2\mathbf{pk}L\int_{T_1}^{T_2} dT$$

Or

$$Q \ln \left(\frac{r_2}{r_1}\right) = 2\mathbf{pk}L(T_1 - T_2)$$
or,

$$Q = \frac{2\mathbf{p}\mathbf{k}L(T_1 - T_2)}{\ln\left(\frac{r_2}{r_1}\right)}$$

If a composite cylindrical wall is involved (e.g. metal pipe clad with layers of insulation)

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$$Q = \frac{-2\mathbf{p}L\Delta T}{\frac{\ln(r_2/r_1)}{\mathbf{k}_1} + \frac{\ln(r_3/r_2)}{\mathbf{k}_2} + \frac{\ln(r_4/r_3)}{\mathbf{k}_3} + \dots}$$

Similar equations can be developed for spherical surfaces.

Example Problem:

A cylindrical pipe with a 5 cm inner diameter is used to carry a hot fluid. The pipe's wall is 5 mm thick and has a thermal conductivity of 25 W/(m.K). Determine the rate of heat loss per unit length (in units of kW/m) if the temperature of the hot fluid is 300° C, and the outer surface of the cylinder remains at the ambient temperature of 25° C.

$$R_1 = 2.5 \text{cm}$$

 $R_2 = 2.5 + 0.5 = 3 \text{cm}$

$$Q = \frac{2\pi \kappa L (T_1 - T_2)}{\ln \left(\frac{r_2}{r_1}\right)}$$

$$\frac{Q}{L} = \frac{2\pi \kappa (T_1 - T_2)}{\ln \left(\frac{r_2}{r_1}\right)} = \frac{2\pi \times 25 \times (300 - 25)}{\ln(3/2.5)} = 236.93 \,\text{kW/m}$$