

* Problem 1

1. 1. We know the following:

$$\text{Aerodynamic drag force, } F_D = \frac{1}{2} \rho V^2 C_D A_f \quad -(1.1.1)$$

$$\text{Power eqn, } F_A \cdot v = P_w = \dot{v} F_D \quad -(1.1.2)$$

↑
Engine @ wheels.

At terminal vel., acc. = 0 $\Rightarrow F_A = 0$

$$\Rightarrow P_w - \dot{v} F_D = 0 \Rightarrow \boxed{F_D = \frac{P_w}{\dot{v}}} \quad -(1.1.3)$$

$$1.2. P_w = n_g \cdot P_e \quad -(1.2.1)$$

$$\text{Power eqn, } F_A \cdot v = P_w - \dot{v} F_D \quad -(1.2.2)$$

At terminal vel., acc = 0 $\Rightarrow F_A = 0$

$$\Rightarrow P_w = \dot{v} F_D = \dot{v} \left[\frac{1}{2} \rho V^2 C_D A_f \right]$$

$$\therefore \boxed{C_D A_f = \frac{2 P_w}{\dot{v} V^3}} \quad -(1.2.3)$$

Acc. to (Wang, 2001), $A_f \approx 79\% \text{to } 84\% \text{ of (width x height)}$

Taking average value, $A_f = \left(\frac{79+84}{2} \right) \% \text{ of (width x height)}$

$$A_f = \frac{81.5}{100} \times \text{width} \times \text{height} \quad -(1.2.4)$$

$$\therefore C_D = \frac{\text{value of (1.2.3)}}{A_f}$$

$$1.3 \cdot \text{Power eq?}, F_A \cdot v = P_w - v F_D \quad -(1.3.1)$$

At terminal vel., acc = 0 $\Rightarrow F_A = 0$

$$\Rightarrow P_w - v F_D = 0$$

$$\Rightarrow P_w - v [1/2 \rho v^2 C_D A_f] = 0$$

$$\Rightarrow C_D = \frac{2 P_w}{\rho v^3 A_f} \quad -(1.3.2)$$

$$1.4 \cdot \text{Power eq?}, F_A \cdot v = P_w - v F_D \quad -(1.4.1)$$

At terminal vel., acc = 0 $\Rightarrow F_A = 0$

$$\Rightarrow P_w - v F_D = 0 \quad -(1.4.2)$$

For achieving same speed "v"

$$P_{w_M} - v F_{D_M} = P_{w_K} - v F_{D_K}$$

\uparrow Mercedes \uparrow Koenigsegg

$$\therefore P_{w_M} = P_{w_K} - v F_{D_K} + v F_{D_M} \quad -(1.4.3)$$

* Problem 2

$$2.C \cdot a_y = \frac{v_{corner}^2}{R} \Rightarrow v_{corner} = \sqrt{R \cdot a_y} \quad -(2.C.1)$$

$$2.E \cdot S_T = \frac{(v_{brake})^2 - (v_{acc})^2 + 2 |a_{brake}| \cdot L}{2(a_{acc} + |a_{brake}|)} \quad -(2.E.1)$$

2.D. max. vel. is at transition point s_T .

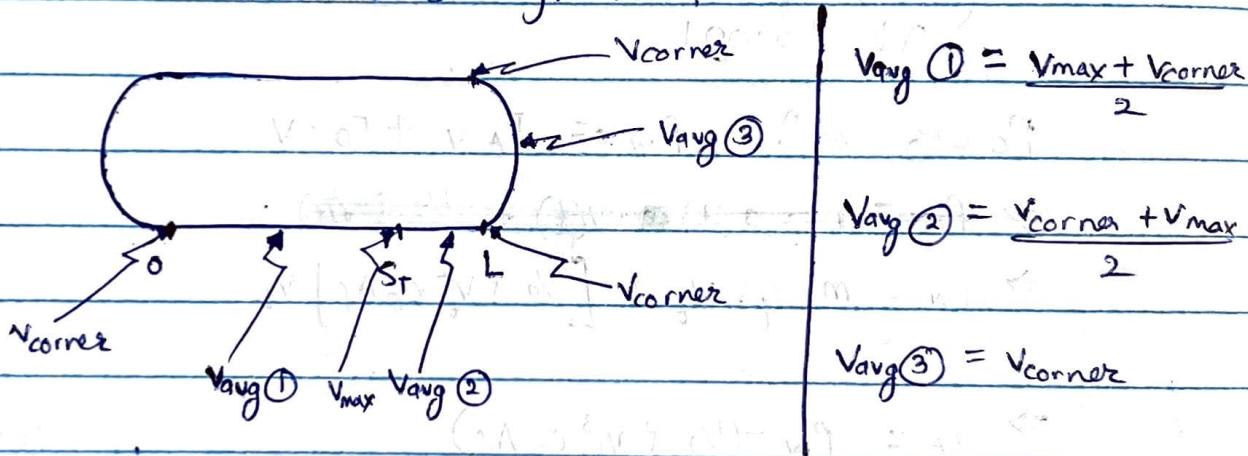
$$V_{\max}^2 = V_{\text{corner}}^2 + 2 \cdot a_{\text{acc}} \cdot s_T$$

$$\therefore V_{\max} = \sqrt{V_{\text{corner}}^2 + 2 \cdot a_{\text{acc}} \cdot s_T}$$

- (2.D.1)

2.B. For const. acc., $V_{\text{avg}} = \frac{V_f + V_i}{2}$ - (2.B.1)

The track has 3 segments:



Note: since the track is symmetrical, avg. vel. in other half is same.

$$V_{\text{avg}} = \frac{V_{\text{avg}}(1) + V_{\text{avg}}(2) + V_{\text{avg}}(3)}{3} = \frac{2 \left(\frac{V_{\max} + V_{\text{corner}}}{2} \right) + V_{\text{corner}}}{3}$$

$$\therefore V_{\text{avg}} = \frac{V_{\max} + 2V_{\text{corner}}}{3}$$

- (2.B.2)

2.A. Since we solved this problem for optimal breaking performance, it is evident that this is the highest avg. vel. possible

$$\Rightarrow t_{\min} = d / V_{\text{avg}}$$

- (2.A.1)

* Problem 3

$$3 \cdot C: \frac{a_y}{R} = v_{\text{corner}}^2 \Rightarrow v_{\text{corner}} = \sqrt{R \cdot a_y} \quad - (3 \cdot C \cdot 1)$$

3 · E: ~~Let $\Delta t = 0.001$~~ Acceleration is variable;
Forward
Start at the begining of straight,

$$\begin{cases} t(0) = 0 \\ v(0) = v_{\text{corner}} \\ s(0) = 0 \\ \Delta t = 0.001 \end{cases}$$

$$\text{Power eq?} \quad P_w = F_A \cdot v + F_D \cdot v$$

$$\Rightarrow P_w = m \cdot a(t) \cdot v(t) + \frac{1}{2} \cdot S \cdot v^2(t)$$

$$\Rightarrow P_w = m \cdot a_t \cdot v_t + \left[\frac{1}{2} S v_t^2 \cdot C_D A_f \right] v_t$$

$$\Rightarrow a_t = \frac{P_w - \left(\frac{1}{2} S v_t^2 C_D A_f \right)}{m \cdot v_t} \quad - (3 \cdot E \cdot 1)$$

Euler forward:

$$\begin{bmatrix} v \\ s \end{bmatrix}_{t+1} = \begin{bmatrix} v \\ s \end{bmatrix}_t + \Delta t \cdot \begin{bmatrix} a_t \\ v_t \end{bmatrix} \quad - (3 \cdot E \cdot 2)$$

solve above $s = [0; L]$

Meanwhile, braking has constant acceleration:

$$(v_{\text{brake}})_t^2 = (v_{\text{corner}})^2 + 2 |a_{\text{brake}}| s_t \quad - (3 \cdot E \cdot 3)$$

After each iteration, check if the forward acc. velocity $(v_{acc})_t$ is equal to the braking velocity ~~$(v_{brake})_t$~~ $(v_{brake})_t$ corresponding to distance s_t as given by eqⁿ (3.E.2) and (3.E.3) respectively.

The point where these velocities match is the transition point, and hence the iterations can be stopped.

The corresponding point in s_t from eqⁿ (3.E.2) gives the distance "s_T" of transition point from start of the straight.

3.D: The max. vel. can be retrieved based on the index of s_T in the s_t array. The velocity in v_t array corresponding to that index is v_{max} .

3.B: As in case of 2.B, we have 3 segments,

(i) $V_{avg} \textcircled{1}$ can be computed by taking the mean of v_t array ; $V_{avg} \textcircled{1} = \frac{\sum_{t=0}^n v_t}{n}$

$$(ii) V_{avg} \textcircled{2} = \frac{V_{corner} + V_{max}}{2}$$

$$(iii) V_{avg} \textcircled{3} = V_{corner}$$

$$\therefore V_{avg} = \underline{\text{mean}(V_t)} + \frac{V_{corner} + V_{max}}{2} + V_{corner} \quad - (3.B.1)$$

3.A. As in case of 2.A.,

$$t_{min} = d / V_{avg}$$

- (3.A.1)

* Problem 4

4.1. Solved programmatically [add V_{wind} to vehicle vel. to account for the headwind]

4.2. Solved manually by looking at the plot.

4.3.

(a) Drag coeff.

$$c_d = \frac{2 \cdot m \cdot \beta \cdot \tan^{-1}(\beta)}{V_0 \cdot T \cdot S \cdot A_f} \quad - (4.3.1)$$

substitute $V_0 = (V_0 + V_{wind})$ to account for headwind.



(b) Rolling resistance,

$$R_x = \frac{V_0 \cdot m \cdot \tan^{-1}(\beta)}{\beta \cdot T} \quad - (4.3.2)$$

substitute $V_0 = (V_0 + V_{wind})$ to account for headwind