

1. (5 points) Show that for any matrix $A \in \mathbb{R}^{n \times n}$, the infinite series $e^A = I + A + \frac{A^2}{2!} + \dots$ converges.

Solution:

The norm of matrix A is defined to be

$$\|A\| = \sup_{x \neq 0} \frac{\|Ax\|}{\|x\|} \quad (1)$$

Equivalently, $\|A\|$ is the smallest number λ such that $\|Ax\| \leq \lambda\|x\|$ for all $x \in \mathbb{R}^n$.

So we know that $\|A\|$ is finite. Furthermore, it is easy to get

$$\|AB\| \leq \|A\|\|B\| \quad (2)$$

which leads

$$\|A^m\| \leq \|A\|^m \quad (3)$$

We know for the space of matrix as $\mathbb{R}^{n \times n}$, every Cauchy sequence converges. And any series which converges absolutely also converges.

So we can get

$$\sum_{m=0}^{\infty} \left\| \frac{A^m}{m!} \right\| \leq \sum_{m=0}^{\infty} \frac{\|A\|^m}{m!} = e^{\|A\|} < \infty \quad (4)$$

which means $e^A = I + A + \frac{A^2}{2!} + \dots$ also converges. \square

2. (5 points) Given a linear system $\dot{x} = Ax + Bu$. Its solution is given by

$$x(t) = e^{At}x_0 + \int_0^t e^{A(t-\tau)}Bu(\tau)d\tau$$

Now assume we have $u(t) \equiv u_k$ for $t \in [k\delta t, (k+1)\delta t)$. Please derive the Zero-order-hold discretization rule, namely, derive expressions for A_d and B_d such that

$$x_{k+1} = A_dx_k + B_d u_k$$

where $x_k \triangleq x(k \cdot \delta t)$ and $u_k = u(k \cdot \delta t)$

Solution: We know

$$x_k = e^{Ak\delta t}x(0) + \int_0^{k\delta t} e^{A(k\delta t-\tau)}Bu(\tau)d\tau \quad (5)$$

and

$$\begin{aligned} x_{k+1} &= e^{A(k+1)\delta t}x(0) + \int_0^{(k+1)\delta t} e^{A((k+1)\delta t-\tau)}Bu(\tau)d\tau \\ &= e^{A\delta t} \left(e^{Ak\delta t}x(0) + \int_0^{k\delta t} e^{A(k\delta t-\tau)}Bu(\tau)d\tau \right) + \int_{k\delta t}^{(k+1)\delta t} e^{A((k+1)\delta t-\tau)}Bu(\tau)d\tau \end{aligned} \quad (6)$$

Substitute (5) into (6) and let $(k+1)\delta t - \tau = \alpha$ we can get

$$\begin{aligned} x_{k+1} &= e^{A\delta t} x_k + \int_0^{\delta t} e^{A\alpha} B u((k+1)\delta t - \alpha) d\alpha \\ &= e^{A\delta t} x_k + \int_0^{\delta t} e^{A\tau} d\tau B u_k \end{aligned} \quad (7)$$

So

$$A_d = e^{A\delta t} \quad B_d = \int_0^{\delta t} e^{A\tau} d\tau B$$

and

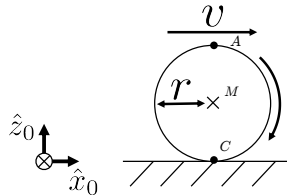
$$\begin{aligned} \int_0^{\delta t} e^{A\tau} d\tau &= \int_0^{\delta t} \left(I + A\tau + \frac{A^2}{2!}\tau^2 \right) d\tau \\ &= \delta t I + \frac{A}{2!}\delta t^2 + \frac{A^2}{3!}\delta t^3 + \dots \end{aligned} \quad (8)$$

□

3. **Spatial Velocity:** (2×6 points) A cylinder rolls without slipping in the \hat{x}_0 direction on the $\hat{x}_0 - \hat{y}_0$ plane. The cylinder has a radius of r and a constant forward speed of v . Let ${}^0C = [C_x(t), 0, 0]^T$ be the position of the contact point at time t . Let ${}^0A = [A_x(t), 0, 0]^T$ be the position of the instantaneous top of the cylinder at time t .

- What is the linear velocity of the point C ? (hint: just need to compute $\frac{d}{dt}C_x(t)$)?
- What is the linear velocity of the point A ?
- What is velocity of the body-fixed point currently coincides with C ?
- What is velocity of the body-fixed point currently coincides with A ?
- What is the spatial velocity of the cylinder in $\{0\}$ -frame?
- What is the spatial velocity of the cylinder in frame $\{C\}$? ($\{C\}$ has the same orientation as $\{0\}$, while its origin is at the contact point C)

Note: The first 4 questions are all referring to the inertia frame $\{0\}$



Solution: (a): The constant forward speed is v . So

$$\frac{dC_x(t)}{dt} = v \quad (9)$$

The linear velocity of the point C is $[v, 0, 0]^T$.

(b): Similarly to (a)

$$\frac{dA_x(t)}{dt} = v \quad (10)$$

The linear velocity of the point A is $[v, 0, 0]^T$.

(c): We can get the angular velocity of the cylinder as

$${}^0\omega = \begin{bmatrix} 0 \\ \frac{v}{r} \\ 0 \end{bmatrix} \quad (11)$$

So the velocity of the body-fixed point currently coincides with C is

$${}^0v_C = \begin{bmatrix} v - |{}^0\omega|r \\ 0 \\ 0 \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \\ 0 \end{bmatrix} \quad (12)$$

$$\text{Or } {}^0v_C = {}^0v_M + {}^0\omega \times {}^0(\overrightarrow{MC}) = [0, 0, 0]^T$$

(d): Similarly to (c), the velocity of the body-fixed point currently coincides with A is

$${}^0v_A = \begin{bmatrix} 2v \\ 0 \\ 0 \end{bmatrix} \quad (13)$$

$$\text{Or } {}^0v_A = {}^0v_M + {}^0\omega \times {}^0(\overrightarrow{MA}) = [2v, 0, 0]^T$$

(e): The velocity of the body-fixed point currently coincides with the origin of frame $\{0\}$ is

$${}^0v_0 = \begin{bmatrix} v - |{}^0\omega|r \\ 0 \\ |{}^0\omega|C_x(t) \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \\ \frac{vC_x(t)}{r} \end{bmatrix} \quad (14)$$

So the spatial velocity of the cylinder in $\{0\}$ -frame is

$${}^0\mathcal{V} = \begin{bmatrix} 0 \\ \frac{v}{r} \\ 0 \\ 0 \\ 0 \\ \frac{vC_x(t)}{r} \end{bmatrix} \quad (15)$$

(f): Similarly to (e), the spatial velocity of the cylinder in frame $\{C\}$ is

$${}^C\mathcal{V} = \begin{bmatrix} 0 \\ \frac{v}{r} \\ 0 \\ 0 \\ 0 \\ 0 \end{bmatrix} \quad (16)$$

□

4. **Spatial Velocity:** (2×8 points) Modern Robotics: Exercise 5.5

Solution: We can construct the twist (in spatial coordinates) for the revolute joint as

$$\mathcal{V} = \begin{bmatrix} \omega \\ v \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \\ 1 \\ -\begin{bmatrix} 0 \\ 0 \\ 1 \end{bmatrix} \times \begin{bmatrix} L \\ L \\ 0 \end{bmatrix} \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \\ 1 \\ L \\ L \\ 0 \end{bmatrix} \quad (17)$$

(a): The position of P is

$$\begin{aligned} {}^sP(t) &= e^{[\mathcal{V}]\theta} {}^sP(0) \\ &= \begin{bmatrix} \cos \theta & -\sin \theta & 0 & L + L \sin \theta - L \cos \theta \\ \sin \theta & \cos \theta & 0 & L - L \sin \theta - L \cos \theta \\ 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 1 \end{bmatrix} \begin{bmatrix} L \\ L \\ 0 \\ 1 \end{bmatrix} \\ &= \begin{bmatrix} L + d \sin \theta \\ L - d \cos \theta \\ 0 \\ 1 \end{bmatrix} \end{aligned} \quad (18)$$

(b): The velocity of point P in terms of the fixed frame is

$$\begin{aligned}
\dot{P} &= [V_s]P(t) \\
&= \dot{\theta} \begin{bmatrix} 0 & -1 & 0 & L \\ 1 & 0 & 0 & -L \\ 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 \end{bmatrix} \begin{bmatrix} L + d \sin \theta \\ L - d \sin \theta \\ 0 \\ 1 \end{bmatrix} \\
&= \begin{bmatrix} d \cos \theta \\ d \sin \theta \\ 0 \\ 0 \end{bmatrix}
\end{aligned} \tag{19}$$

or

$$\dot{P} = \frac{d^s P(t)}{dt} = \begin{bmatrix} d \cos \theta \\ d \sin \theta \\ 0 \\ 0 \end{bmatrix} \tag{20}$$

(c):

$$\begin{aligned}
T_{sb}(t) &= e^{[\mathcal{V}]t} T_{sb}(0) \\
&= \begin{bmatrix} \cos \theta & -\sin \theta & 0 & L + d \sin \theta \\ \sin \theta & \cos \theta & 0 & L - d \cos \theta \\ 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 1 \end{bmatrix}
\end{aligned} \tag{21}$$

(d): The twist of T_{sb} in body coordinates is

$$\mathcal{B} = \begin{bmatrix} 0 \\ 0 \\ 1 \\ d \\ 0 \\ 0 \end{bmatrix} \tag{22}$$

(e): The twist of T_{sb} in spatial coordinates is

$$\mathcal{V} = \begin{bmatrix} 0 \\ 0 \\ 1 \\ L \\ -L \\ 0 \end{bmatrix} \tag{23}$$

(f):

$$\begin{aligned}
\mathcal{V} &= \begin{bmatrix} \cos \theta & -\sin \theta & 0 & 0 & 0 & 0 \\ \sin \theta & \cos \theta & 0 & 0 & 0 & 0 \\ 0 & 0 & 1 & 0 & 0 & 0 \\ 0 & 0 & L - d \cos \theta & \cos \theta & -\sin \theta & 0 \\ 0 & 0 & -L - d \sin \theta & \sin \theta & \cos \theta & 0 \\ L(\sin \theta - \cos \theta) + d & L(\cos \theta + \sin \theta) & 0 & 0 & 0 & 1 \end{bmatrix} \mathcal{B} \\
&= \begin{bmatrix} R & 0 \\ [p]R & R \end{bmatrix} \mathcal{B} \\
&= [\text{Ad}_T] \mathcal{B}
\end{aligned} \tag{24}$$

(g): Let

$$\mathcal{B} = \begin{bmatrix} \omega^b \\ v^b \end{bmatrix} \tag{25}$$

We have

$${}^a R_b^T \dot{P} = v^b \tag{26}$$

And we also have

$$T_{sb}^{-1} \dot{P} = [\mathcal{B}]^b P(t) \tag{27}$$

(h): Let

$$\mathcal{V} = \begin{bmatrix} \omega \\ v \end{bmatrix} \tag{28}$$

We have

$$-[\omega]^s P(t) + \dot{P} = v \tag{29}$$

And we also have

$$\dot{P} = [\mathcal{V}]^s P(t) \tag{30}$$

□

5. Screw axis and its transformation: (3 × 3 points)

- (a) Draw the screw axis for the twist $\mathcal{V} = (0, 2, 2, 4, 0, 0)$
- (b) Consider an arbitrary screw axis \mathcal{S} . Suppose the axis has gone through a rigid body transformation $T = (R, p)$ and the resulting new screw axis is \mathcal{S}' . Show that

$$\mathcal{S}' = [\text{Ad}_T] \mathcal{S}$$

(we have given the proof in class, you need to go through it on your own again)

- (c) Consider a rigid body motion: rotation about z axis counterclockwise by 90° and then translate along negative y -axis by 1m. All the axes are with respect to the fixed inertia frame.
- Find the numerical values of the corresponding transformation matrix T ;
 - Move the screw axis in part (a) using T . Find the new screw axis \mathcal{S}' after the motion.

Solution: (a): The axis is

$$l = \left\{ \frac{\omega \times v}{\|\omega\|^2} + \lambda \omega : \lambda \in \mathbb{R} \right\}$$

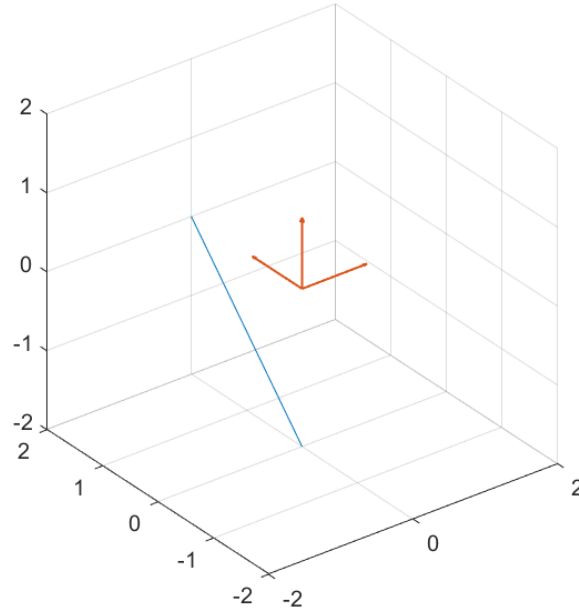


Figure 1: Axis for \mathcal{V}

(b): We define the original frame is $\{A\}$ and there is a screw axis \mathcal{S} in $\{A\}$. After the transformation (a rigid motion) $\{A\}$ becomes $\{B\}$, the corresponding screw axis is \mathcal{S}' . But the coordinate represented in their own frame is the same which is

$${}^A\mathcal{S} = {}^B\mathcal{S}' \quad (31)$$

We already know there exist a twist corresponding to a screw motion. In some sense we can regard it as a “same” thing which means the adjoint transformation for change of coordinates

can be applied on a screw axis. With slight abuse of notation, left multiply $[\text{Ad}_{A_{T_B}}]$ in (31)

$$[\text{Ad}_{A_{T_B}}]^A \mathcal{S} = [\text{Ad}_{A_{T_B}}]^B \mathcal{S}' \quad (32)$$

So we have

$$[\text{Ad}_{A_{T_B}}]^A \mathcal{S} = {}^A \mathcal{S}' \quad (33)$$

Omit superscript and subscript

$${}^A \mathcal{S}' = [\text{Ad}_T] \mathcal{S} \quad (34)$$

(c):

(i):

$$T = \begin{bmatrix} 0 & -1 & 0 & 0 \\ 1 & 0 & 0 & -1 \\ 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 1 \end{bmatrix} \quad (35)$$

(ii):

$$\mathcal{S}' = [\text{Ad}_T] \mathcal{S} = \begin{bmatrix} R & 0 \\ [p]R & R \end{bmatrix} \mathcal{S} = \begin{bmatrix} -2 \\ 0 \\ 2 \\ -2 \\ 4 \\ -2 \end{bmatrix} \quad (36)$$

The screw coordinates of \mathcal{S}' is

$$\begin{aligned} \text{Pitch:} \quad h &= \frac{\omega^T v}{\|\omega\|^2} = 0 \\ \text{Axis:} \quad l &= \{[-1, -1, -1]^T + \lambda[-2, 0, 2] : \lambda \in \mathbb{R}\} \\ \text{Magnitude:} \quad M &= 2\sqrt{2} \end{aligned} \quad (37)$$

See Fig. 2.

□

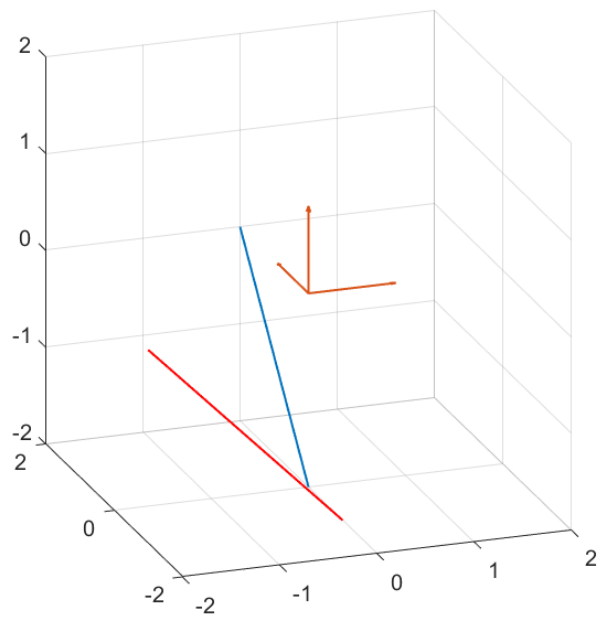


Figure 2: Screw axis after transformation