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Shigley's Mechanical Engineering Design | (10th Edition)

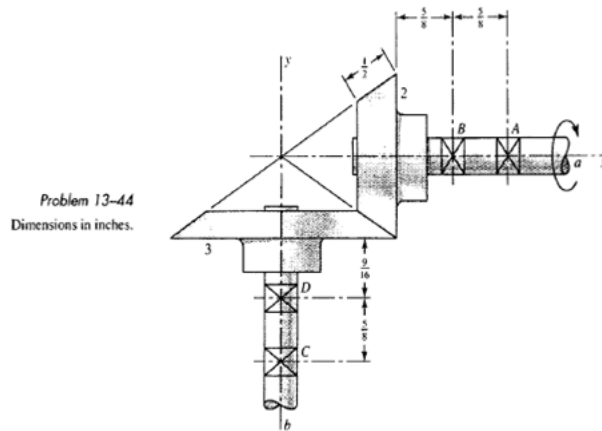
Chapter 13, Problem 44P

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Problem

The figure shows a 10 diametral pitch 18-tooth 20 straight bevel pinion driving a 30-tooth gear. The transmitted load is 25 lbf. Find the bearing reactions at *C* and *D* on the output shaft if *D* is to take both radial and thrust loads.



Step-by-step solution

Step 1 of 12

Draw the free body diagram of the gear set.

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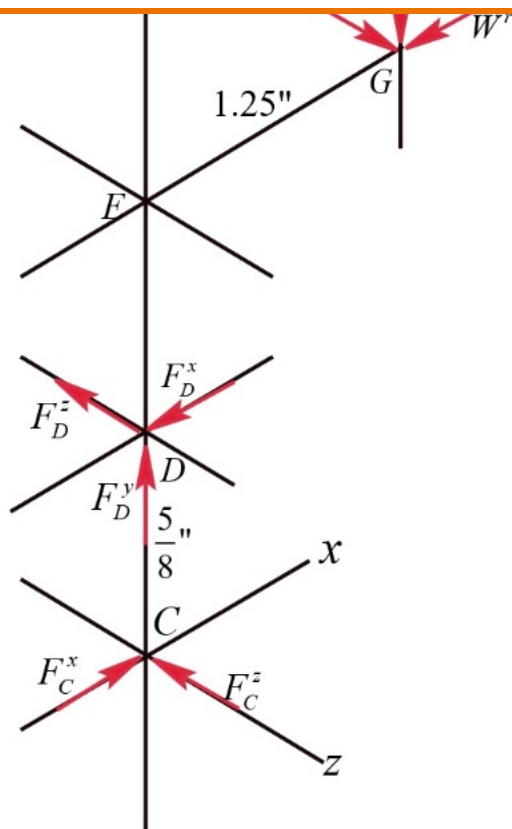


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Step 2 of 12

Calculate the diameter of gear 2.

$$d_2 = \frac{N_2}{p_d}$$

Here, N_2 is the number of teeth on gear 2 and p_d is the diametric pitch.

Substitute 18 teeth for N_2 and 10 teeth/in for p_d .

$$\begin{aligned} d_2 &= \frac{18}{10} \\ &= 1.8 \text{ in} \end{aligned}$$

Calculate the diameter of gear 3.

$$d_3 = \frac{N_3}{p_d}$$

Here, N_3 is the number of teeth on gear 3.

Substitute 30 teeth for N_3 and 10 teeth/in for p_d .

$$\begin{aligned} d_3 &= \frac{30}{10} \\ &= 3 \text{ in} \end{aligned}$$

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Step 3 of 12

Calculate the pitch angles.



Substitute 1.8 in for d_2 and 3 in for d_3 .

$$\Gamma = 180^\circ - \gamma$$

Substitute 30.96° for γ .

$$\begin{aligned}\Gamma &= 180^\circ - 30.96^\circ \\ &= 59.04^\circ\end{aligned}$$

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Step 4 of 12

Calculate the distance between D and E.

$$DE = \frac{9}{16} + 0.5 \cos \Gamma$$

Substitute 59.04° for Γ .

$$\begin{aligned}DE &= \frac{9}{16} + 0.5 \cos 59.04^\circ \\ &= 0.8197 \text{ in}\end{aligned}$$

Calculate the radial load.

$$W^r = W^t \tan \phi \cos \Gamma$$

Here, the transmitted load is W^t and the pressure angle is ϕ .

Substitute 25 lbf for W^t , 20° for ϕ , and 59.04° for Γ .

$$\begin{aligned}W^r &= 25 \tan(20^\circ) \cos(59.04^\circ) \\ &= 4.681 \text{ lbf}\end{aligned}$$

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Step 5 of 12

Calculate the total load.

$$W^a = W^t \tan \phi \sin \Gamma$$

Here, the transmitted load is W^t and the pressure angle is ϕ .

Substitute 25 lbf for W^t , 20° for ϕ , and 59.04° for Γ .

$$\begin{aligned}W^a &= 25 \tan(20^\circ) \sin(59.04^\circ) \\ &= 7.803 \text{ lbf}\end{aligned}$$

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Step 6 of 12

Write the load vector on the gear.

$$\mathbf{W} = -W^r \mathbf{i} - W^a \mathbf{j} + W^t \mathbf{k}$$

Substitute 4.681 lbf for W^r , 7.803 lbf for W^a , and 25 lbf for W^t .

$$\mathbf{W} = -4.681 \mathbf{i} - 7.803 \mathbf{j} + 25 \mathbf{k}$$

Write the position vector between D and G.

$$\mathbf{R}_{DG} = 0.8197 \mathbf{j} + 1.25 \mathbf{i}$$

Write the position vector between D and C.

$$\mathbf{R}_{DC} = -0.625 \mathbf{j}$$

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Calculate the moment about point D.

$$\sum M_D = 0$$

$$\mathbf{R}_{DG} \times \mathbf{W} + \mathbf{R}_{DC} \times \mathbf{F}_C + \mathbf{T} = 0$$

Substitute $0.8197\mathbf{j} + 1.25\mathbf{i}$ for \mathbf{R}_{DG} , $-4.68\mathbf{i} - 7.803\mathbf{j} + 25\mathbf{k}$ for \mathbf{W} , $-0.625\mathbf{j}$ for \mathbf{R}_{DC} , $F_C^x\mathbf{i} + F_C^z\mathbf{k}$ for \mathbf{F}_C , and $T\mathbf{j}$ for \mathbf{T} .

$$(0.8197\mathbf{j} + 1.25\mathbf{i}) \times (-4.68\mathbf{i} - 7.803\mathbf{j} + 25\mathbf{k}) + (-0.625\mathbf{j}) \times (F_C^x\mathbf{i} + F_C^z\mathbf{k}) + T\mathbf{j} = 0$$

$$(20.49\mathbf{i} - 31.25\mathbf{j} - 5.917\mathbf{k}) + (-0.625F_C^x\mathbf{i} + 0.625F_C^z\mathbf{k}) + T\mathbf{j} = 0$$

$$(20.49 - 0.625F_C^x)\mathbf{i} + (-31.25 + T)\mathbf{j} + (-5.917 + 0.625F_C^z)\mathbf{k} = 0$$

Equate \mathbf{i} , \mathbf{j} , and \mathbf{k} components to zero.

$$20.49 - 0.625F_C^x = 0$$

$$F_C^x = 32.784 \text{ lbf}$$

$$-31.25 + T = 0$$

$$T = 31.25 \text{ lbf} \cdot \text{in}$$

$$-5.917 + 0.625F_C^z = 0$$

$$F_C^z = 9.4672 \text{ lbf}$$

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Step 8 of 12

Calculate the bearing reaction at C.

$$F_C = \sqrt{(F_C^x)^2 + (F_C^z)^2}$$

Substitute 9.4672 lbf for F_C^x and 32.784 lbf for F_C^z .

$$F_C = \sqrt{(9.4672)^2 + (32.784)^2}$$

$$= 34.123 \text{ lbf}$$

Therefore, the bearing reaction at C is $\boxed{34.123 \text{ lbf}}$.

[Comment](#)

Step 9 of 12

Apply force equilibrium for the system.

$$\sum \mathbf{F} = 0$$

$$\mathbf{W} + \mathbf{F}_C + \mathbf{F}_D = 0$$

Substitute $-4.68\mathbf{i} - 7.803\mathbf{j} + 25\mathbf{k}$ for \mathbf{W} , $9.4672\mathbf{i} + 32.784\mathbf{k}$ for \mathbf{F}_C , and $F_D^x\mathbf{i} + F_D^y\mathbf{j} + F_D^z\mathbf{k}$ for \mathbf{F}_D .

$$-4.68\mathbf{i} - 7.803\mathbf{j} + 25\mathbf{k} + 9.4672\mathbf{i} + 32.784\mathbf{k} + F_D^x\mathbf{i} + F_D^y\mathbf{j} + F_D^z\mathbf{k} = 0$$

$$(-4.681 + 9.4672 + F_D^x)\mathbf{i} + (-7.803 + F_D^y)\mathbf{j} + (25 + 32.784 + F_D^z)\mathbf{k} = 0$$

Equate \mathbf{i} , \mathbf{j} , and \mathbf{k} components to zero.

$$-4.681 + 9.4672 + F_D^x = 0$$

$$F_D^x = -4.7862 \text{ lbf}$$

$$-7.803 + F_D^y = 0$$

$$F_D^y = 7.803 \text{ lbf}$$

$$25 + 32.784 + F_D^z = 0$$

$$F_D^z = -57.784 \text{ lbf}$$



Step 10 of 12

Therefore, the bearing reactions at D in x , y , and z directions are $\boxed{-4.7862 \text{ lbf}}$, $\boxed{7.803 \text{ lbf}}$, and $\boxed{-57.784 \text{ lbf}}$ respectively.

[Comment](#)

Step 11 of 12

Calculate the radial load at D .

$$F_{D(\text{radial})} = \sqrt{(F_D^x)^2 + (F_D^z)^2}$$

Substitute -4.7862 lbf for F_D^x and -57.784 lbf for F_D^z .

$$\begin{aligned} F_{D(\text{radial})} &= \sqrt{(-4.7862)^2 + (-57.784)^2} \\ &= 57.98 \text{ lbf} \end{aligned}$$

Therefore, the radial load at D is $\boxed{57.98 \text{ lbf}}$.

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Step 12 of 12

Calculate the thrust load at D .

$$F_{D(\text{thrust})} = F_D^y$$

Substitute 7.803 lbf for F_D^y .

$$F_{D(\text{thrust})} = 7.803 \text{ lbf}$$

Therefore, the thrust load at D is $\boxed{7.803 \text{ lbf}}$.

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Recommended solutions for you in Chapter 13

Chapter 13, Problem 38P

For the countershaft in Prob. 3?72, p. 138, assume the gear ratio from gear B to its mating gears 2 to 1. (a) Determine the minimum number of teeth that can be used on gear B without an interference problem in the teeth. (b) Using the number of...

[See solution](#)

Chapter 13, Problem 7P

A parallel helical gearset consists of a 19-tooth pinion driving a 57-tooth gear. The pinion has a left-hand helix angle of 30° , a normal pressure angle of 20° , and a normal module of 2.5 mm. Find: (a) The normal, transversal and axial circular...

[See solution](#)