

## ESS 411/511 Geophysical Continuum Mechanics Class #4

Highlights from Class #3 – Andrew Gregovich

Today's highlights on Friday – Madeleine Lucas

Warm-up question (break-out) –

- What does Quality  $Q$  mean?
- What is it useful for?

Class-prep answers (break-out)

- Discuss your explanations of energy dissipation (or not) in cyclical springs and dash-pots.



## ESS 411/511 Geophysical Continuum Mechanics Class #3

### For Friday class

- Please read Mase, Smelser, and Mase, CH 2 through Section 2.3

Your short CR/NC Pre-class prep writing assignment (1 point) in Canvas

- It will be due in Canvas at the start of class.
- I will send another message when it is posted in Canvas.

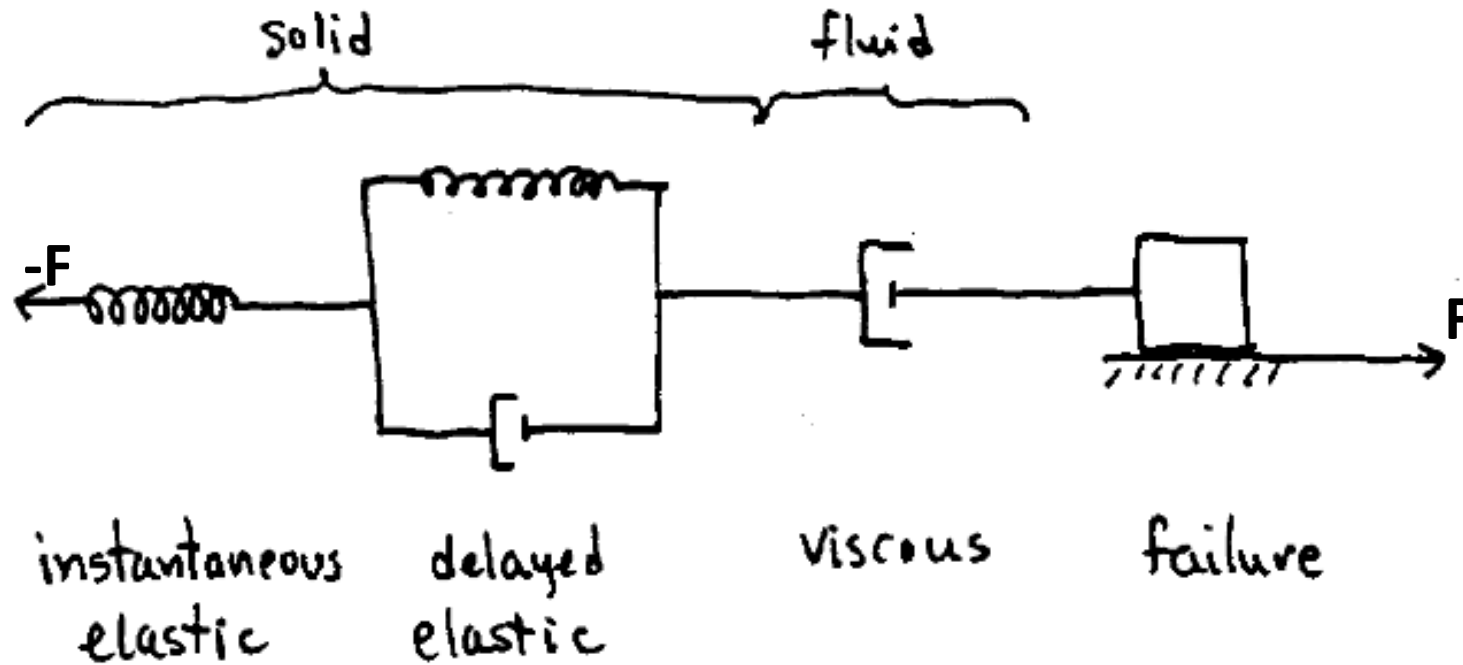
## ESS 411/511 Geophysical Continuum Mechanics Class #3

### Broad Outline for the Quarter

- Continuum mechanics in 1-D
- 1-D models with springs, dashpots, sliding blocks
- Mathematical tools – vectors, tensors, coordinate changes
- Stress – principal values, Mohr's circles for 3-D stress
- Coulomb failure, pore pressure, crustal strength
- Measuring stress in the Earth
- Strain – Finite strain; infinitesimal strains
- Moments – lithosphere bending; Earthquake moment magnitude
- Conservation laws
- Constitutive relations for elastic and viscous materials
- Elastic waves; kinematic waves

## ESS 411/511 Geophysical Continuum Mechanics Class #3

### A model for idealized real materials



Forces are balanced

- Each element feels the same force  $F$

# Rheological tests

## Creep tests

- Apply a constant stress  $\sigma$   
e.g. put a weight on top of a sample
- Measure strain  $e(t)$  or strain rate  $\dot{e}(t)$

## Relaxation tests

- Apply an abrupt strain  $e$ , then hold it constant  
e.g. abrupt shortening in a vice.
- Measure stress  $\sigma(t)$  as sample adjusts.

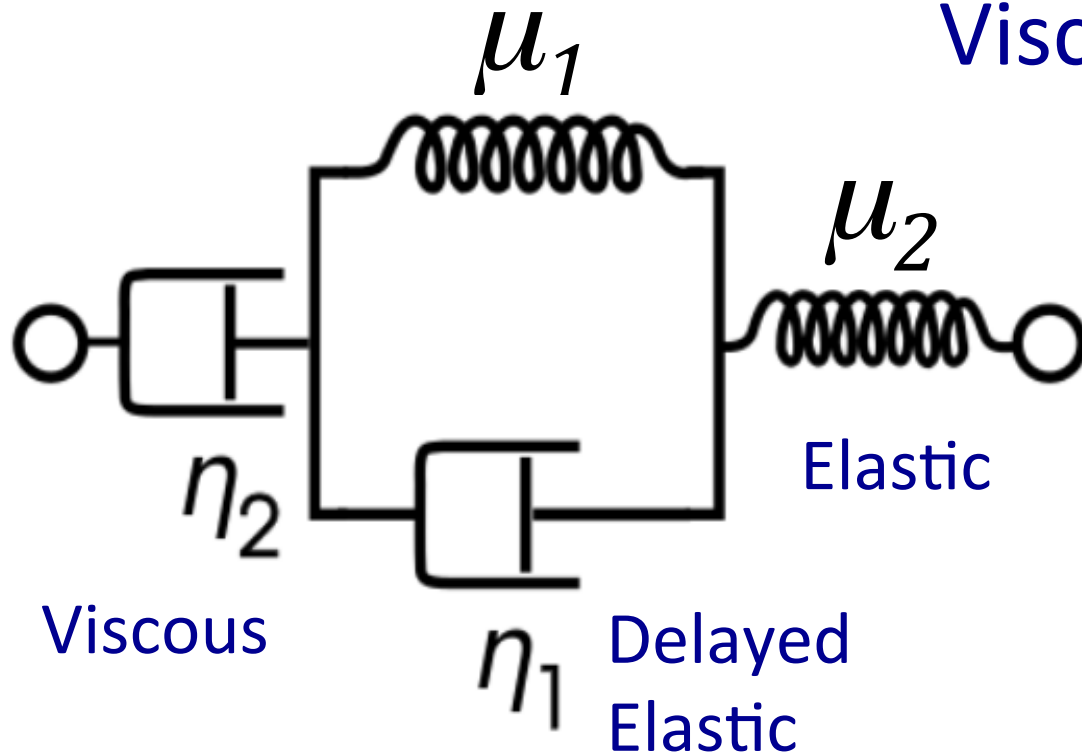
## Constant strain-rate tests

- Apply a constant strain rate  
e.g. with a motor-driven vice
- Measure stress  $\sigma(t)$

# Models for linear solids

Those springs and dashpots ...

## Viscoelastic model



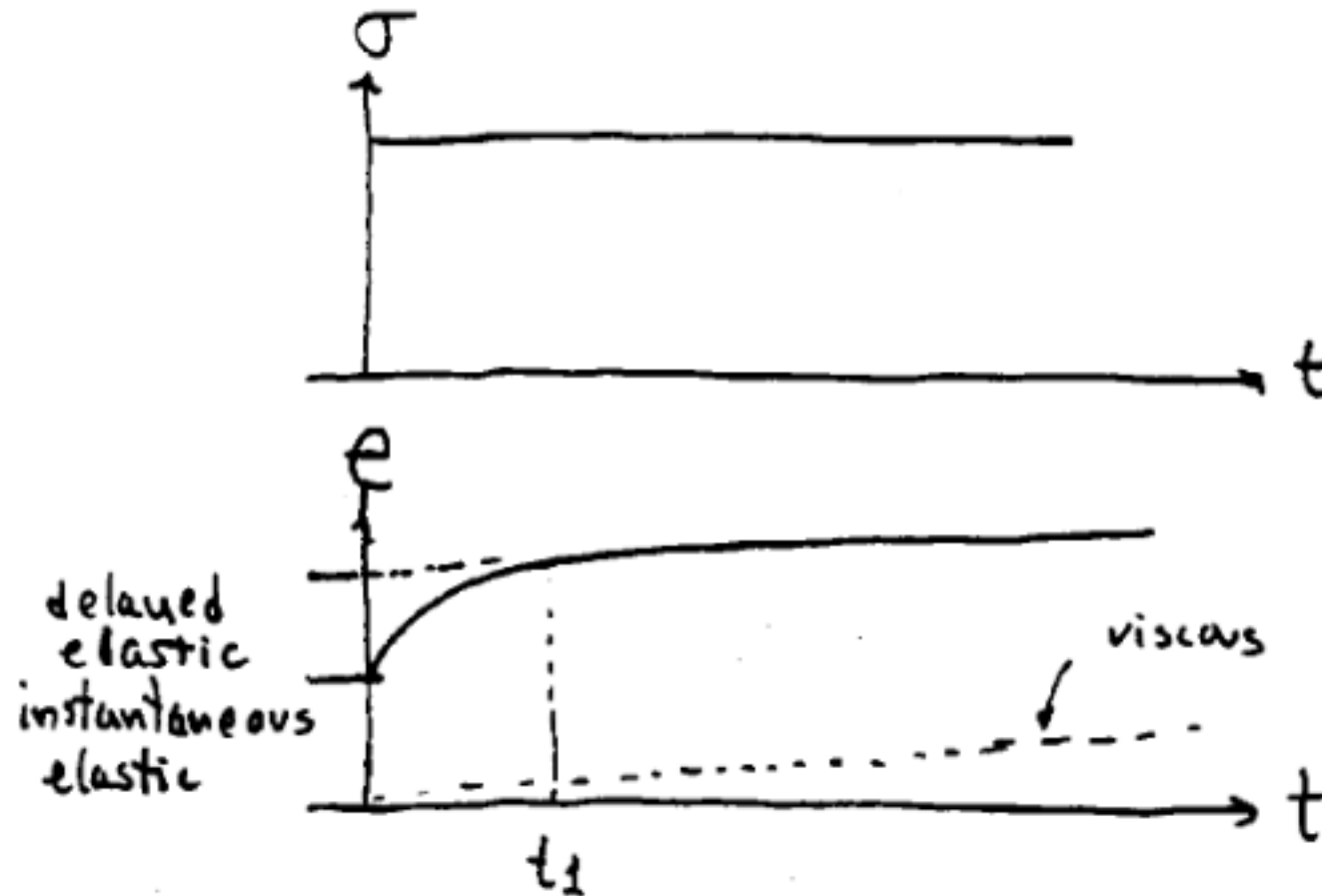
Called *Maxwell Solid*, if  $\eta_1 = \infty$ ,  $\mu_1 = \infty$

Called *Kelvin-Voigt Solid*, if  $\eta_2 = \infty$ ,  $\mu_2 = \infty$

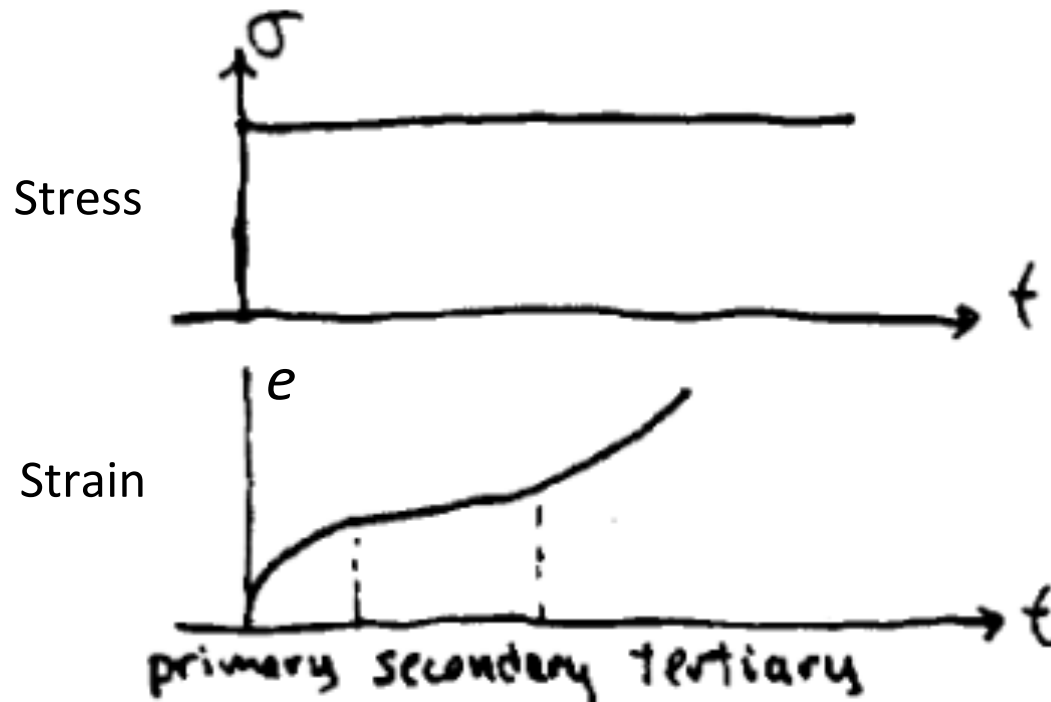
Called *Standard Linear Solid*, if  $\eta_2 = \infty$



# Creep Test with Viscoelastic model



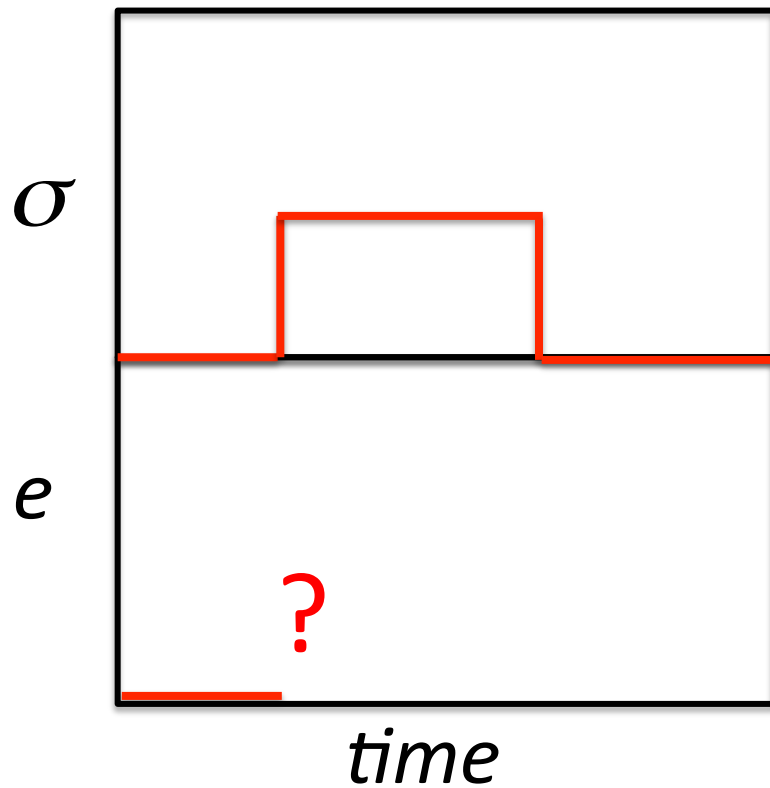
# Viscoelastic behavior in real materials



Changes in the microstructure at the crystal level inside the sample can alter the effective viscosity after significant strain as the test progresses.

- e.g. crystal basal planes align for easy glide
- microcracks may develop, allowing internal slip

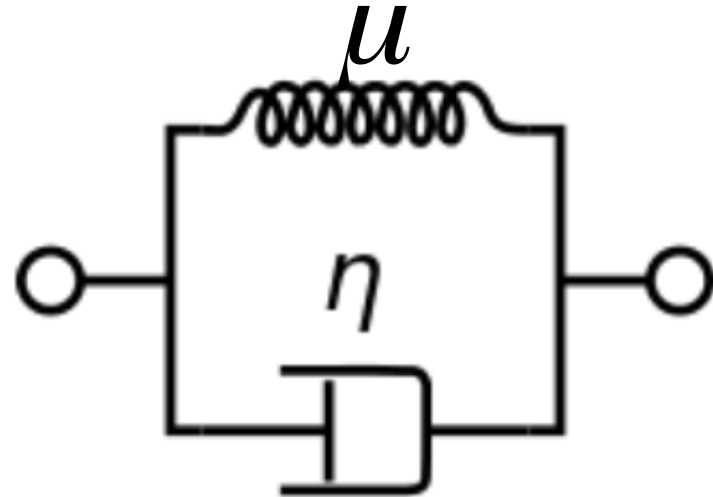
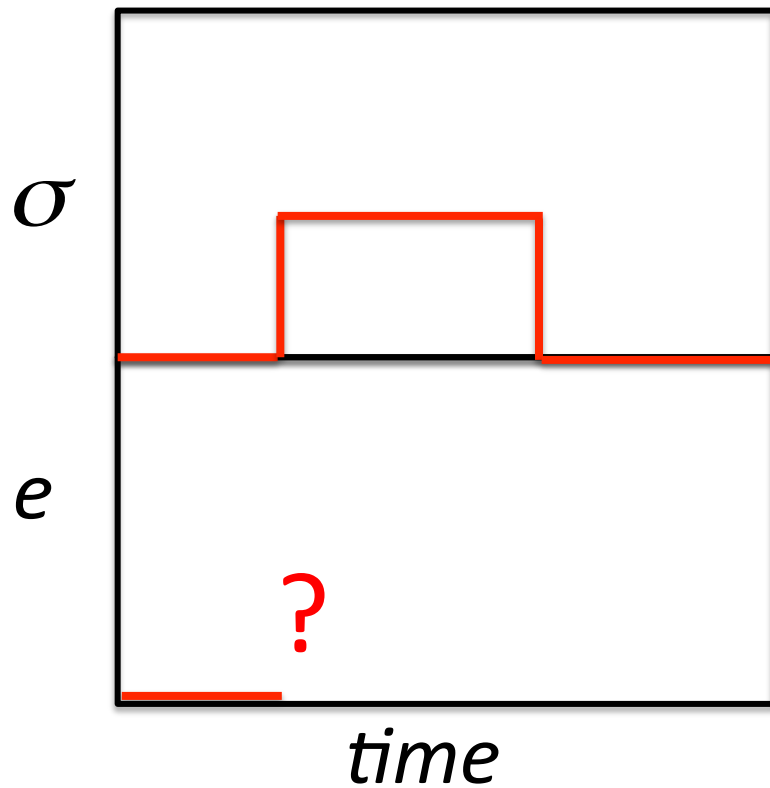
## Maxwell solid



Also Homework set #1

- Is there a characteristic time for the material?
- $\eta/\mu$  ?

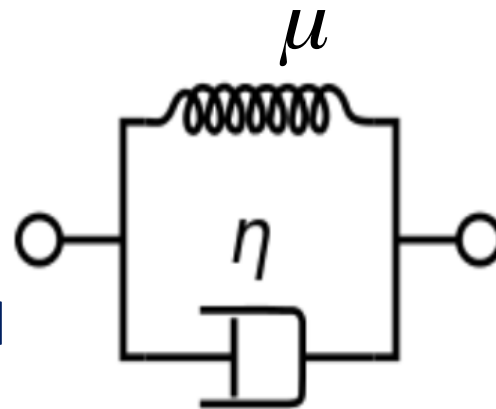
## Kelvin-Voigt solid



- Is there a characteristic time for the material?
- $\eta/\mu$  ?

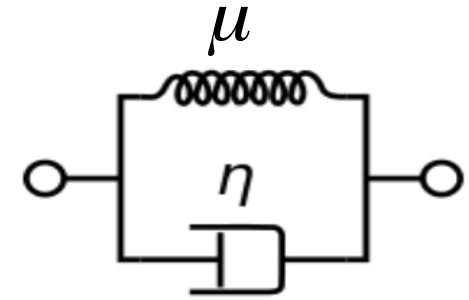


A shock absorber can be modeled as a delayed elasticity Kelvin-Voigt solid



# Kelvin-Voigt Response

Spring and dashpot together support stress  $\sigma$



$$\sigma(t) = \mu e(t) + \eta \dot{e}(t)$$

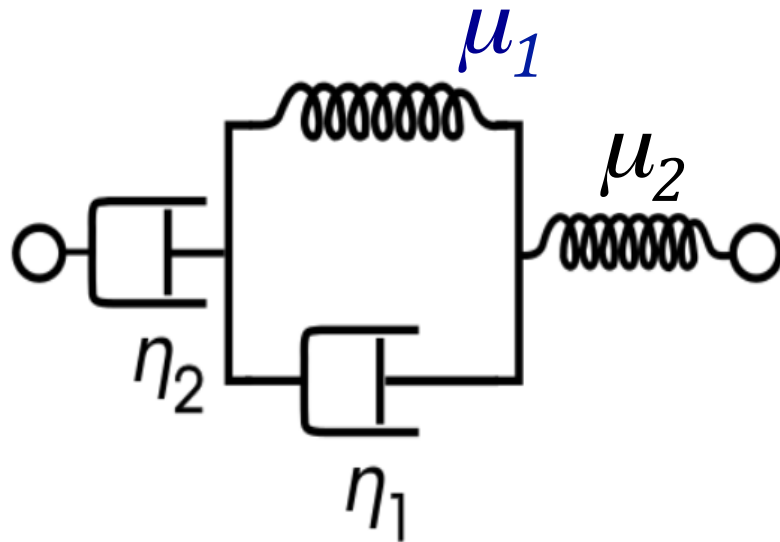
- At  $t = 0$ , spring hasn't shortened; dashpot supports all the stress  $\sigma$ , so  $e(0) = 0$  (\*)
- At  $t = \infty$ , dashpot has stopped; spring supports all the stress  $\sigma$ , so  $e(\infty) = \sigma/\mu$  (\*\*)
- The transition is probably a decaying exponential.
- $\tau = \eta/\mu$  must be the time constant defining the transition.

$$e(t) = \frac{\sigma}{\mu} + A \exp\left(-\frac{\eta}{\mu} t\right)$$

With the boundary conditions (\*) and (\*\*), A can be found, and solution is ...

$$e(t) = \frac{\sigma}{\mu} \left[ 1 - \exp\left(-\frac{\mu}{\eta} t\right) \right]$$

## Generalized linear viscoelastic solid



Response to  
constant loading  $\sigma$

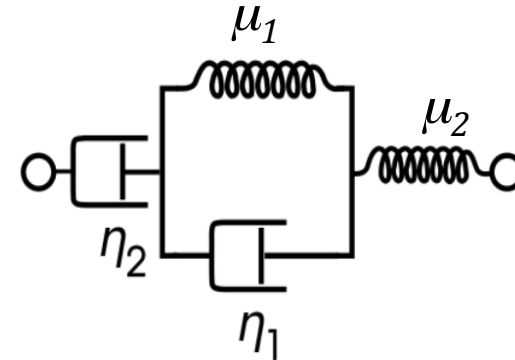
$$e(t) = \frac{\sigma}{\eta_2}t + \frac{\sigma}{\mu_2} + \frac{\sigma}{\mu_1} \left[ 1 - \exp\left(-\frac{\mu_1}{\eta_1}t\right) \right]$$

Viscous

Elastic

Delayed  
Elastic

How did we get that?!



$$e(t) = \frac{\sigma}{\eta_2}t + \frac{\sigma}{\mu_2} + \frac{\sigma}{\mu_1} \left[ 1 - \exp \left( -\frac{\mu_1}{\eta_1}t \right) \right]$$

Viscous

Elastic

Delayed  
Elastic

Each element feels the same stress  $\sigma$ ,

- We just added up the strains in each element



The Raymond notes also give creep functions and relaxation functions for step changes in stress or strain

Creep  
test

$$\sigma(t) = \begin{cases} 0 & t < 0 \\ 1 & t \geq 0 \end{cases} \quad \sigma'(t) = \delta(t) \quad \text{Applied stress } \sigma$$

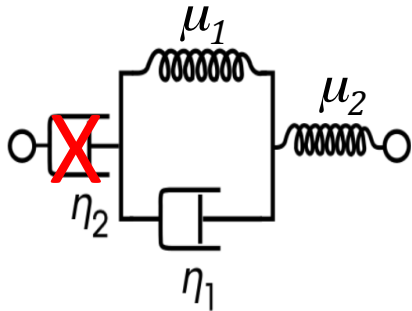
$$e(t) = \int_0^t C(t-t')\delta(t')dt' = C(t) \quad C(t-t') \text{ is the creep function}$$

Relaxation  
test

$$e(t) = \begin{cases} 0 & t < 0 \\ 1 & t \geq 0 \end{cases} \quad e'(t) = \delta(t) \quad \text{Applied strain } \sigma$$

$$\sigma(t) = \int_0^t k(t-t')\delta(t')dt' = k(t) \quad k(t-t') \text{ is the relaxation function}$$

## Relaxation Function in Standard Linear Solid ( $\eta_2 = \infty$ )



$e$  = constant (after  $t=0$ )

$$\sigma(t) = e \frac{\mu_1 \mu_2}{\mu_1 + \mu_2} \left\{ 1 + \frac{\mu_2}{\mu_1} \exp\left(-\frac{\mu_1 + \mu_2}{\eta_1} t\right) \right\} (*)$$

**At  $t=0$  :**

- The spring  $\mu_1$  in the K-V element is prevented from deforming, due to  $\eta_1$ .
- All applied strain  $e$  is taken up initially in the spring  $\mu_2$ . So  $\sigma(0) = \mu_2 e$   
(Do you agree that (\*) shows this?)
- Stress  $\sigma(0)$  also acts on the K-V element, so it also begins to strain.

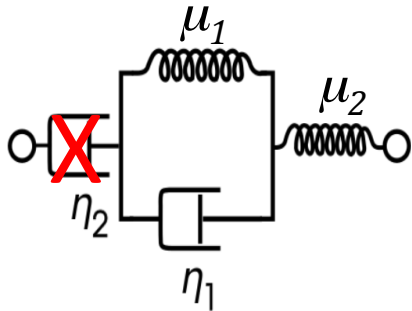
For a K-V element, 
$$e(t) = \frac{\sigma}{\mu} \left[ 1 - \exp\left(-\frac{\mu}{\eta} t\right) \right] \quad (\text{strain } e(0) = 0 \quad \checkmark).$$

By differentiating with respect to time  $t$ ,

$$\dot{e}(t) = \frac{\sigma}{\mu} \left[ \left( \frac{\mu}{\eta} \right) \exp\left(-\frac{\mu}{\eta} t\right) \right] = \frac{\sigma}{\eta} \exp\left(-\frac{\mu}{\eta} t\right)$$

At  $t=0$ , the strain *rate* in the K-V element is 
$$\dot{e}(0) = \frac{\sigma}{\eta} = \frac{\mu_2 e}{\eta}$$

## Relaxation Function in Standard Linear Solid ( $\eta_2 = \infty$ )



$e = \text{constant}$

$$\sigma(t) = e \frac{\mu_1 \mu_2}{\mu_1 + \mu_2} \left\{ 1 + \frac{\mu_2}{\mu_1} \exp\left(-\frac{\mu_1 + \mu_2}{\eta_1} t\right) \right\}^{(*)}$$

### At $t > 0$

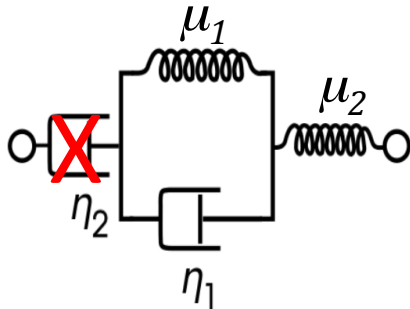
- The K-V element is starting to strain at the rate  $\dot{e}(0) = \frac{\mu_2 e}{\eta}$ ,
- K-V begins to take over some of the strain from the spring  $\mu_2$ .
- Strain  $e_1$  increases in spring  $\mu_1$  and  $\eta_1$ , and strain  $e_2$  decreases in spring  $\mu_2$

$$e_1 + e_2 = e$$

- Because strain is decreasing in spring  $\mu_2$ , stress  $\sigma(t)$  must be decreasing.
- Strain  $e_1$  in spring  $\mu_1$  cannot exceed  $\sigma_\infty / \mu_1$
- Dash-pot  $\eta_1$  must eventually stop moving.
- This means there is no stress in the dash-pot at  $t = t_\infty$
- There will be a time constant  $\tau$  that depends on  $\mu_2$ ,  $\mu_1$  and  $\eta_1$

$$\tau = \left[ \frac{\eta_1}{\mu_1 + \mu_2} \right]$$

## Relaxation Function in Standard Linear Solid ( $\eta_2 = \infty$ )



$e = \text{constant}$

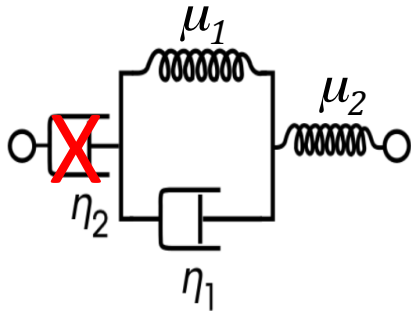
$$\sigma(t) = e \frac{\mu_1 \mu_2}{\mu_1 + \mu_2} \left\{ 1 + \frac{\mu_2}{\mu_1} \exp\left(-\frac{\mu_1 + \mu_2}{\eta_1} t\right) \right\}^{(*)}$$

**At  $t = \infty$**

- Strain  $e_1$  in spring  $\mu_1$  cannot exceed  $\sigma_\infty / \mu_1$
- Dash-pot  $\eta_1$  must eventually stop moving.
- This means there is no stress in the dash-pot at  $t = t_\infty$
- Both springs  $\mu_1$  and  $\mu_2$  then support the same stress  $\sigma_\infty$ , so

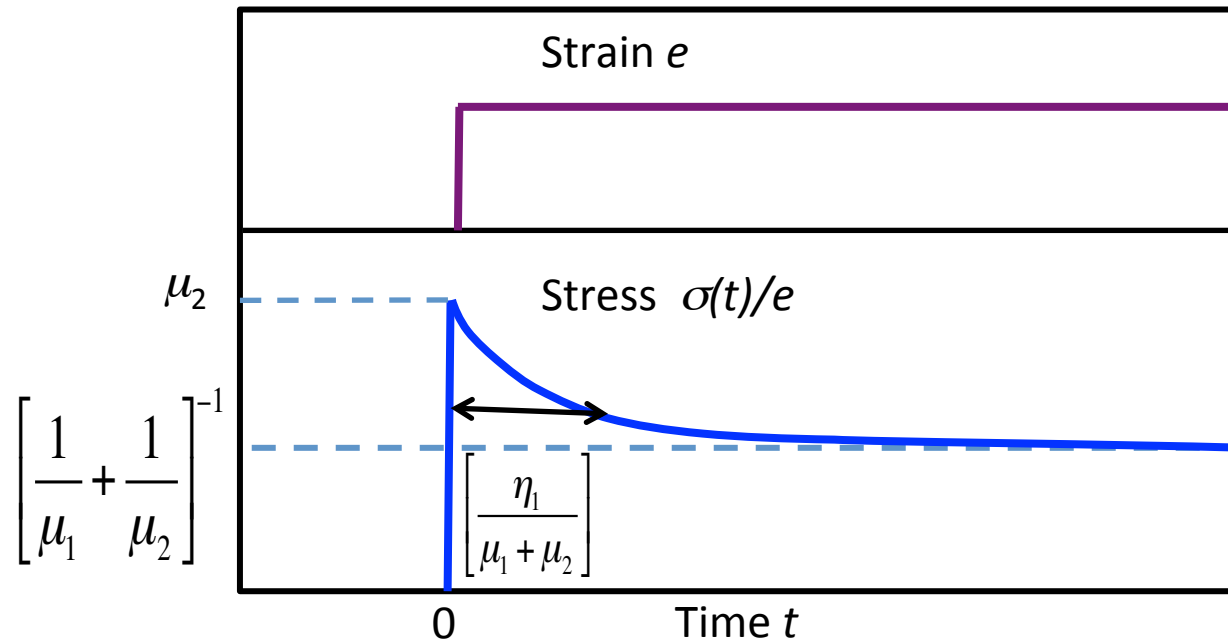
$$e = \frac{\sigma_\infty}{\mu_1} + \frac{\sigma_\infty}{\mu_2} \quad \text{or} \quad \frac{\sigma_\infty}{e} = \left[ \frac{1}{\mu_1} + \frac{1}{\mu_2} \right]^{-1} \quad \text{is the limiting stress at } t = t_\infty$$

## Relaxation Function in Standard Linear Solid ( $\eta_2 = \infty$ )



$e = \text{constant}$

$$\sigma(t) = e \frac{\mu_1 \mu_2}{\mu_1 + \mu_2} \left\{ 1 + \frac{\mu_2}{\mu_1} \exp\left(-\frac{\mu_1 + \mu_2}{\eta_1} t\right) \right\}^{(*)}$$



## Energy and Work

Work  $W$  is force  $\mathbf{F}$  acting through a distance  $d$

Work for point particles:  $W = \mathbf{F} d$

In Continuum – work done per unit volume:

$$\frac{W}{V} = \frac{Fd}{V} = \left( \frac{F}{A} \right) \cdot \left( \frac{d}{l} \right) = \sigma e = \text{stress} \times \text{strain}$$

*Rate of doing work per unit volume*

$$\frac{d}{dt} \left( \frac{W}{V} \right) = \frac{\dot{W}}{V} = \frac{F\dot{d}}{V} = \left( \frac{F}{A} \right) \cdot \left( \frac{\dot{d}}{l} \right) = \sigma \dot{e} = \text{stress} \times \text{strain rate}$$

(overdots indicate time derivatives)

## Energy and Work

Total energy input between from time 0 to  $t$

$$\Delta E(t) = \int_0^t \sigma(t') \dot{e}(t') dt'$$

For elastic material, substitute:  $\sigma(t) = \mu e(t)$

$$\Delta E(t) = \int_0^t \mu e(t') \dot{e}(t') dt' = \frac{1}{2} \mu e^2(t) = \frac{\sigma^2(t)}{2\mu}$$

$\Delta E(t)$  returns to zero whenever  $\sigma$  returns to zero.

- All energy is recovered

## Energy and Work

Total energy input between from time 0 to  $t$

$$\Delta E(t) = \int_0^t \sigma(t') \dot{\epsilon}(t') dt'$$

For viscous  
material:

$$\sigma(t) = \eta \dot{\epsilon}(t)$$

$$\Delta E(t) = \int_0^t \eta \dot{\epsilon}(t')^2 dt'$$

The integrand is always positive.

- $\Delta E(t)$  can never return to zero if strain rate is ever nonzero
- energy is always lost if any strain has occurred.