

Trigonometry

Sine law: $\frac{\sin A}{\sin B} = \frac{a}{b}$ Cosine law: $c^2 = a^2 + b^2 - 2ab \cos C$

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Vectors

$\mathbf{F} = \mathbf{F}_x + \mathbf{F}_y + \mathbf{F}_z = F_x \mathbf{i} + F_y \mathbf{j} + F_z \mathbf{k} = F(\cos \alpha \mathbf{i} + \cos \beta \mathbf{j} + \cos \gamma \mathbf{k})$

$\cos^2 \alpha + \cos^2 \beta + \cos^2 \gamma = 1$

Position vector: $\mathbf{r}_{AB} = (x_B - x_A)\mathbf{i} + (y_B - y_A)\mathbf{j} + (z_B - z_A)\mathbf{k}$

Unit position vector: $\mathbf{u} = \frac{\mathbf{r}}{||\mathbf{r}||}$

$\mathbf{F} = F\mathbf{u}$ (\mathbf{u} is in the direction of \mathbf{F})

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Dot Product

$\langle \mathbf{A}, \mathbf{B} \rangle = \mathbf{A} \cdot \mathbf{B} = A_x B_x + A_y B_y + A_z B_z = AB \cos \theta$

$\cos \theta = \frac{\mathbf{A} \cdot \mathbf{B}}{AB}$

Vector directed along a line: $A_{||} = A \cos \theta = \mathbf{A} \cdot \mathbf{u}$

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Determinants

Second order:

$$\begin{vmatrix} a_1 & a_2 \\ b_1 & b_2 \end{vmatrix} = a_1 b_1 - a_2 b_2$$

Third order:

$$\begin{vmatrix} a_1 & a_2 & a_3 \\ b_1 & b_2 & b_3 \\ c_1 & c_2 & c_3 \end{vmatrix} = a_1 \begin{vmatrix} b_2 & b_3 \\ c_2 & c_3 \end{vmatrix} - a_2 \begin{vmatrix} b_1 & b_3 \\ c_1 & c_3 \end{vmatrix} + a_3 \begin{vmatrix} b_1 & b_2 \\ c_1 & c_2 \end{vmatrix}$$

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Cross Product

Magnitude of cross product: $C = AB \sin \theta$

$\mathbf{C} = \mathbf{A} \times \mathbf{B} = (AB \sin \theta)\mathbf{u_c}$ $\mathbf{u_c} \perp \mathbf{A}, \mathbf{B}$ by the RHR

$$\mathbf{A} \times \mathbf{B} = \begin{vmatrix} \mathbf{i} & \mathbf{j} & \mathbf{k} \\ A_x & A_y & A_z \\ B_x & B_y & B_z \end{vmatrix}$$

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Moments

$M = Fd$ (scalar form) $\mathbf{M} = \mathbf{r} \times \mathbf{F}$ (vector form)

Moment about a specified axis:

- Scalar form: $M = Fd$
- Vector form: $\mathbf{M_a} = [\mathbf{u_a} \cdot (\mathbf{r} \times \mathbf{F})]\mathbf{u_a}$

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Shear Force and Bending Moment Diagrams (SFD & BMD)

Shear and load: $\frac{dV}{dx} = -w(x)$ $\int_a^b dV = \int_a^b -w(x)dx$

Moment and shear: $\frac{dM}{dx} = V(x)$ $\int_a^b dM = \int_a^b V(x)dx$

Concentrated loads: +ve if point load \uparrow -ve if point load \downarrow

Couple moments: +ve if \odot (CW) -ve if \ominus (CCW)

Friction

Max static friction force: $F_{\max} = \mu_s N$

Kinetic friction force: $F = \mu_k N$

Angle of static friction: $\tan \phi_s = \frac{F_s}{N} = \mu_s$

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Center of Gravity and Mass

Center of Gravity (Uniform Body)

$$\bar{x} = \frac{\int \tilde{x} dW}{\int dW} \quad \bar{y} = \frac{\int \tilde{y} dW}{\int dW} \quad \bar{z} = \frac{\int \tilde{z} dW}{\int dW}$$

Center of Gravity (Non-Uniform Body)

$$\bar{x} = \frac{\int \tilde{x} \gamma dV}{\int \gamma dV} \quad \bar{y} = \frac{\int \tilde{y} \gamma dV}{\int \gamma dV} \quad \bar{z} = \frac{\int \tilde{z} \gamma dV}{\int \gamma dV}$$

$\gamma = \rho g = \frac{mg}{V} = \frac{W}{V} \quad \Rightarrow \quad dW = \gamma dV$

Center of Mass (Uniform Body)

$$\bar{x} = \frac{\int \tilde{x} dm}{\int dm} \quad \bar{y} = \frac{\int \tilde{y} dm}{\int dm} \quad \bar{z} = \frac{\int \tilde{z} dm}{\int dm}$$

Center of Mass (Non-Uniform Body)

$$\bar{x} = \frac{\int \tilde{x} \rho dV}{\int \rho dV} \quad \bar{y} = \frac{\int \tilde{y} \rho dV}{\int \rho dV} \quad \bar{z} = \frac{\int \tilde{z} \rho dV}{\int \rho dV}$$

$\rho = \frac{m}{V} \quad \Rightarrow \quad dm = \rho dV$

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Centroids

Centroid of a Volume

$$\bar{x} = \frac{\int_V \tilde{x} dV}{\int_V dV} \quad \bar{y} = \frac{\int_V \tilde{y} dV}{\int_V dV} \quad \bar{z} = \frac{\int_V \tilde{z} dV}{\int_V dV}$$

Centroid of an Area

$$\bar{x} = \frac{\int_A \tilde{x} dA}{\int_A dA} \quad \bar{y} = \frac{\int_A \tilde{y} dA}{\int_A dA} \quad \bar{z} = \frac{\int_A \tilde{z} dA}{\int_A dA}$$

Centroid of a Line

$$\bar{x} = \frac{\int_L \tilde{x} dL}{\int_L dL} \quad \bar{y} = \frac{\int_L \tilde{y} dL}{\int_L dL} \quad \bar{z} = \frac{\int_L \tilde{z} dL}{\int_L dL}$$

Centroids by Composite Bodies

$$\bar{x} = \frac{\sum \tilde{x}W}{\sum W} \quad \bar{y} = \frac{\sum \tilde{y}W}{\sum W} \quad \bar{z} = \frac{\sum \tilde{z}W}{\sum W}$$

Set $A < 0$ for holes.

section	\tilde{x}	A	$\tilde{x}A$
\vdots	\vdots	\vdots	\vdots
Sum	n/a	\cdots	\cdots

Area Moments of Inertia

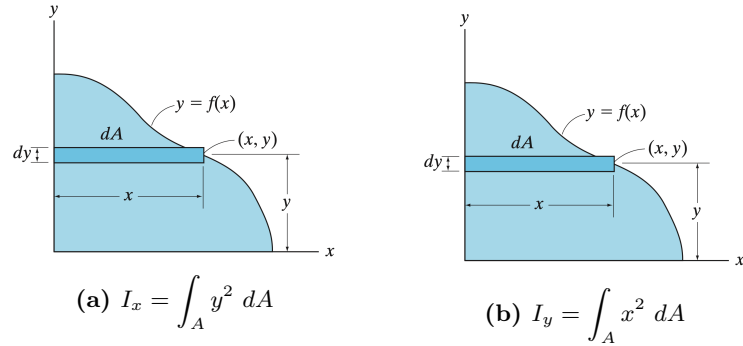


Figure 1: Length of dA should be *parallel* to axis about which I is computed.

Polar moment of inertia: $J_0 = I_x + I_y = \int_A r^2 dA$

Product moment of inertia: $I_{xy} = \int_A xy dA$

Radii of gyration:

$$k_x = \sqrt{\frac{I_x}{A}} \quad k_y = \sqrt{\frac{I_y}{A}} \quad k_0 = \sqrt{\frac{I_0}{A}}$$

Parallel axis theorem:

$$\begin{aligned} I_x &= \bar{I}_{x'} + Ad_y^2 & I_y &= \bar{I}_{y'} + Ad_x^2 \\ J_0 &= \bar{J}_c + Ad^2 & I_{xy} &= \bar{I}_{x'y'} + Ad_x d_y \end{aligned}$$

Area Moments of Inertia by Composite Bodies

Find the appropriate centroids. Set $A < 0$ for holes.

section	$\tilde{I}_{x'}$	$d = \tilde{x} - x$	A	$\bar{I}_x = Ad^2$
\vdots	\vdots	\vdots	\vdots	\vdots
Sum	\cdots	n/a	n/a	\cdots

Geometric Properties of Line and Area Elements

Centroid Location	Centroid Location	Area Moment of Inertia
<p>Circular arc segment</p>	<p>Circular sector area</p>	$I_x = \frac{1}{4} r^4 \left(\theta - \frac{1}{2} \sin 2\theta \right)$ $I_y = \frac{1}{4} r^4 \left(\theta + \frac{1}{2} \sin 2\theta \right)$
<p>Quarter and semicircle arcs</p>	<p>Quarter circle area</p>	$I_x = \frac{1}{16} \pi r^4$ $I_y = \frac{1}{16} \pi r^4$
<p>Trapezoidal area</p>	<p>Semicircular area</p>	$I_x = \frac{1}{8} \pi r^4$ $I_y = \frac{1}{8} \pi r^4$
<p>Semiparabolic area</p>	<p>Circular area</p>	$I_x = \frac{1}{4} \pi r^4$ $I_y = \frac{1}{4} \pi r^4$
<p>Exparabolic area</p>	<p>Rectangular area</p>	$I_x = \frac{1}{12} b h^3$ $I_y = \frac{1}{12} h b^3$
<p>Parabolic area</p>	<p>Triangular area</p>	$I_x = \frac{1}{36} b h^3$