

FINITE ELEMENT TECHNIQUES(FALL 2018)

Final Project

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Objective: -Q4 and Q8 element implementation for a cantilever beam with end load in MATLAB for different mesh sizes under plane stress state and comparing stresses and deflection with Ansys results.

Implementation in MATLAB: -

1) Plane stress analysis of solid using linear Q4 elements

Variable descriptions

k = element matrix

f = element vector

kk = system matrix

ff = system vector

displ = system nodal displacement vector

eldisp = element nodal displacement vector

gcoord = coordinate values of each node

nodes = nodal connectivity of each element

index = a vector containing system dofs associated with each element

bcdof = a vector containing dofs associated with boundary conditions

bcval = a vector containing boundary condition values associated with the dofs in 'bcdof'

q=shear stress equation

ForceNodes=nodes on which load is acting

ShapFun=Shape functions matrix;

DiffMatr=Differentiation matrix

Jacobian=jacobian matrix

TransMatrx=Transformation matrix

Bmtx2='B' matrix

DisplX=Displacement along X,DisplY=Displacement along Y

NodeStrain=Nodal strains

NodeStress=nodal stresses

ElemeStrains=Element strains

ElemeStresses=Element stresses

eleStrain.m file consists of element strain calculation.

Input data:-D=1; Load=10000, E=2e11, Length=5, Poisson ratio=0.3

Procedure: -

Meshing with Element sizes of 0.3,0.4,0.7

1) Initializing all the input variables like number of elements, number of nodes, number of degrees of freedom etc.

2) Importing the nodal coordinates, mesh data and force nodes from Ansys through excel File named 'Data.xlsx'. Then we have find the loads on the nodes using the shear stress and shape functions matrix.

3)Ux, Uy equations are used for specifying the displacement boundary conditions.

4) Calculating a Constitutive matrix for plain stress analysis.

Here in command window you are prompted to enter Desired gaussian quadrature points 1,2 or 3

5)finding Jacobean, transformation matrix, differentiation matrix and B matrix

6) Then Calculating element stiffness matrix using 2X2 ,3,1-point gaussian quadrature;

7) Assembling of the elements.

8) Finding the system matrix.

9) Then finding out stress at different nodes and elements.

Ansys for very fine Mesh-Q4 element

Preprocessing

Step 1. Define Element Type

ANSYS Main Menu: Preprocessor → Element Type → Add/Edit/Delete

In the pop-up window (Element Types), select Add. Then in the new window (Library of Element Types), select Structural Mass → Solid, Quad 4 node 182. OK to close.

In options, element behavior, pick 'plane stress'.

Step 2. Define Material Type

(1) ANSYS Main Menu: Preprocessor → Material Props → Material Models → Structural (double click) → Linear → Elastic → Isotropic

Input Mat'l Properties {EX=Young's '2. e11'; PRXY=0.3; OK}

Step3. Define Geometry

(1) Ansys Main Menu: Preprocessor → Modeling → Create → Areas → Rectangle → By 2 Corners
Enter X = 0; WP Y = -0.5; Width = 5; Height = 1, Click Apply to create rectangle.

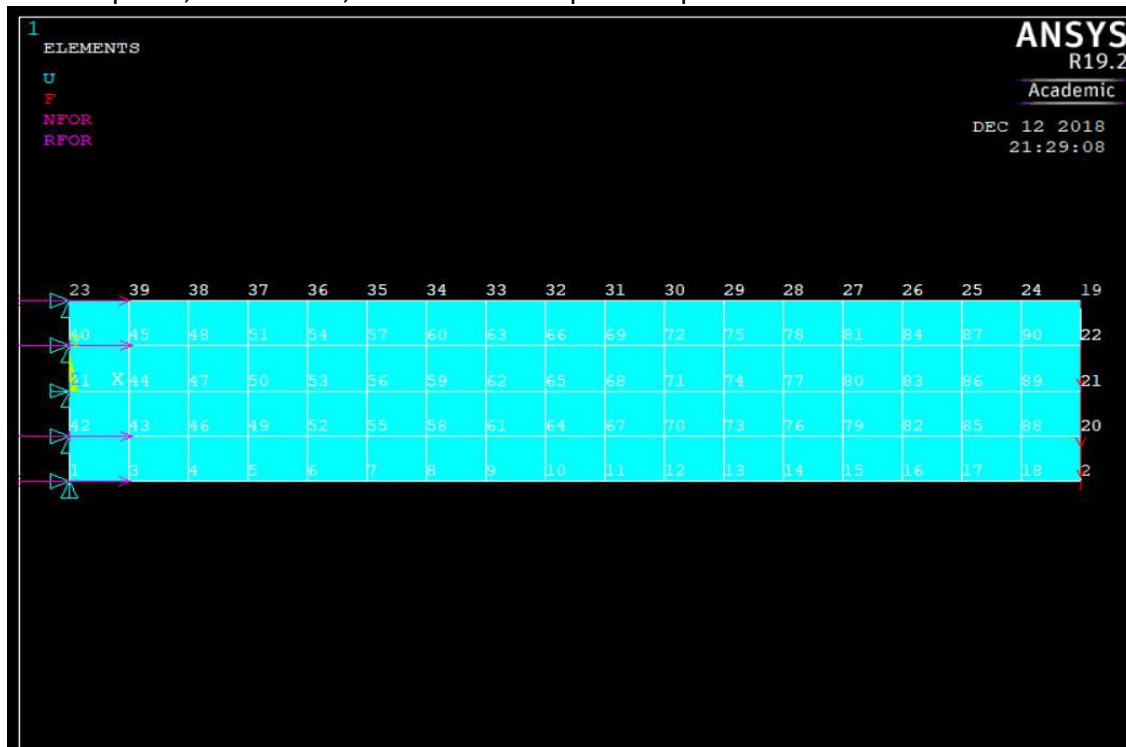
Step 4. Create Mesh

Ansyes Main Menu: Preprocessor → Meshing → Mesh Tool

A new "Mesh Tool" Window opens. Select Global.

In Size Controls panel, set global element SIZE as 0.3.

In Mesh panel, select Area, click Mesh. And pick the plate area to mesh



Solving

Step 6. Apply Displacement Constraints:

Solve->Define Loads->Apply->Functions>

First define the displacement equations of UX, UY given and save the file.

Then Read the files UY.fun, UY.fun and names as DisplX and DisplY correspondingly.

Ansyes Main Menu: Solution → Define Loads → Apply → Structural → Displacement → On nodes

Pick the left edge, select UX->instead of 'Constant value' use 'Existing Table' ->then select 'DisplX'. Repeat the same for UY also.

Step 7. Apply Loads

For Loads first find the individual nodal in MATLAB by using sigma-xy and shape functions

Ansys Main Menu: Solution → Define Loads → Apply → Structural → Force/Moment → On nodes

->pick->top node and give $F_y = -468.8$, similarly for bottom node $F_y = -468.8$. node-22=-2500,node-21=-3437.5,Node-20=-2500

Step 8. Solve

Ansys Main Menu: Solution → Solve → Current LS

Close Solution Done Window.

Postprocessing

Step 9. Plot Deformed Shape

ANSYS Main Menu: General Postproc → Plot Results → Deformed Shape

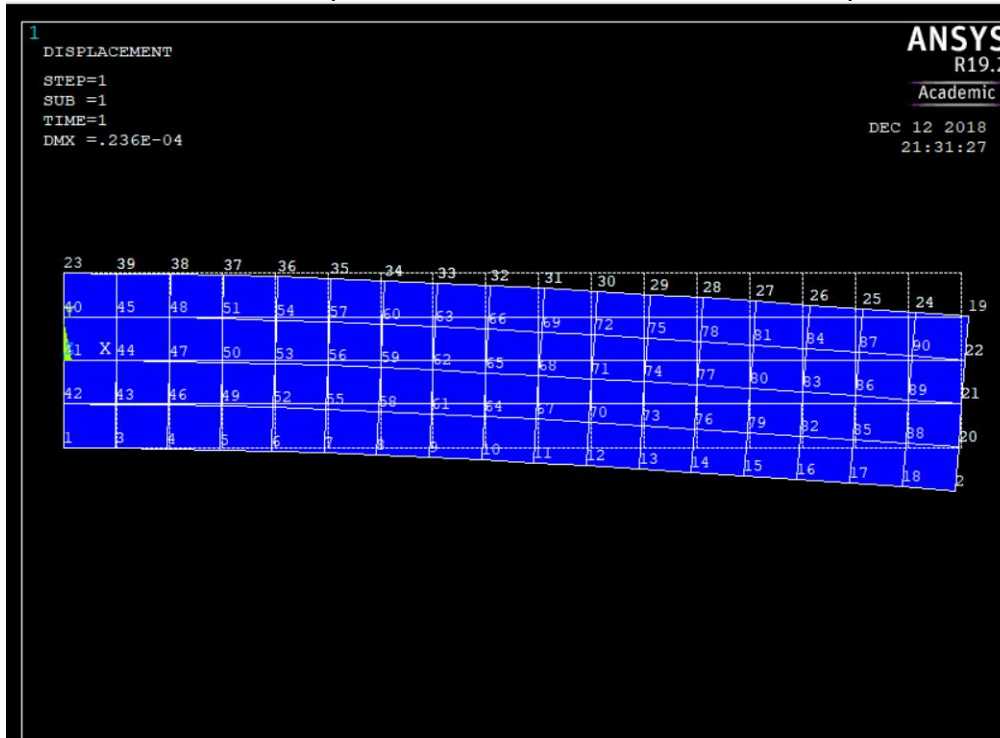
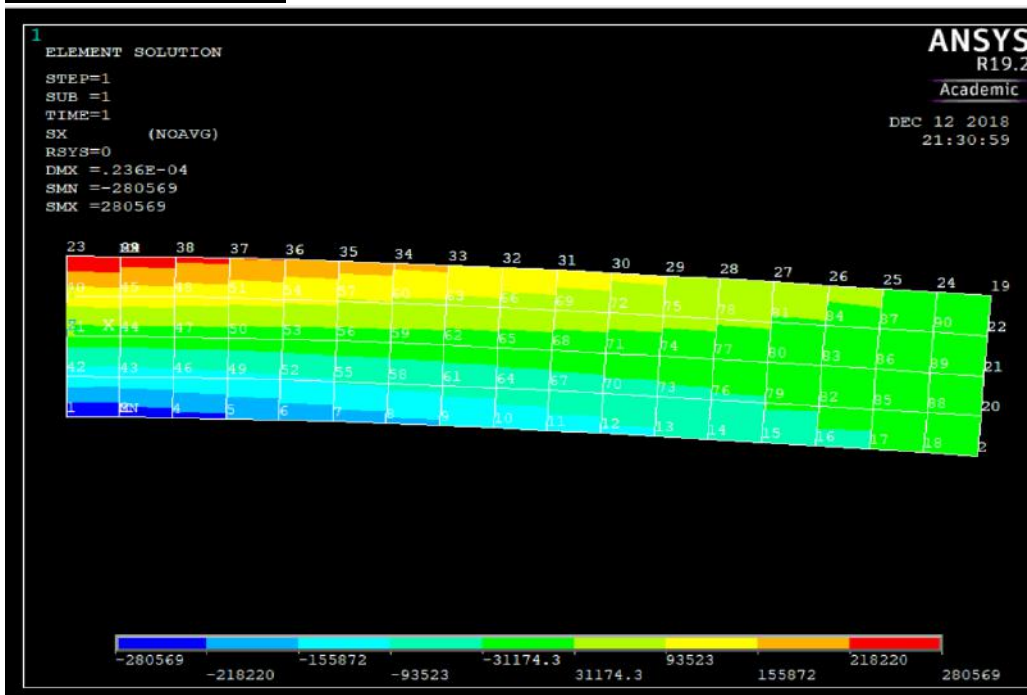


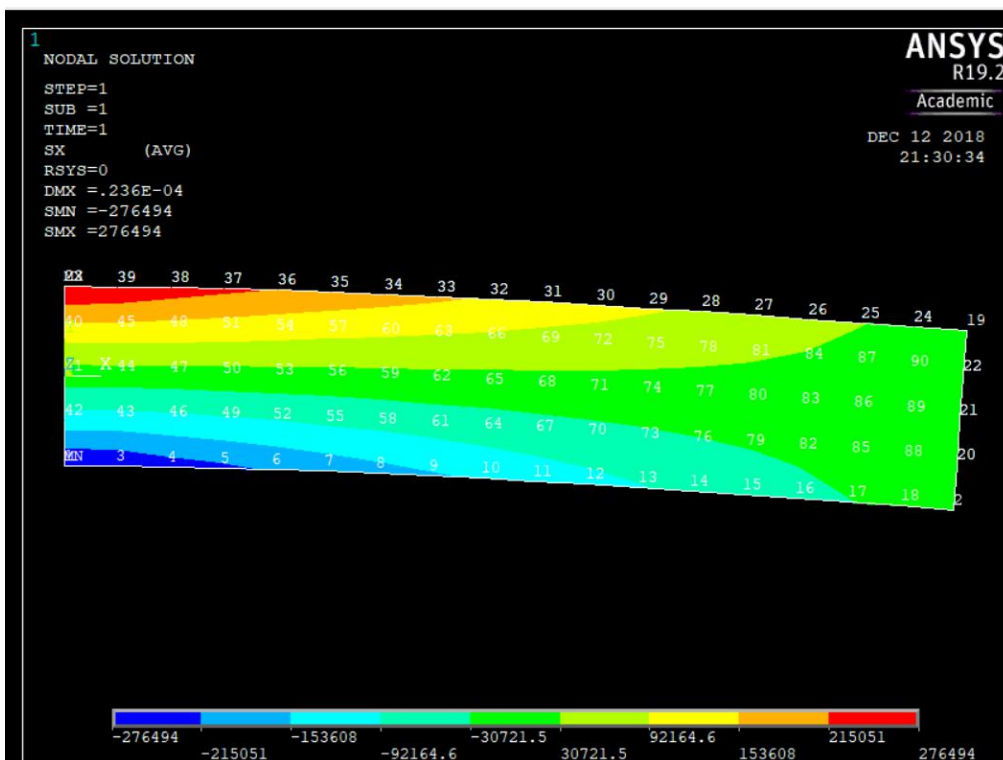
Fig.3 Deformed shape overlaid on outline of undeformed shape

Step 10. Contour Plots of Stresses ANSYS Main Menu: General Postproc → Plot Results → Contour Plot
→Element Solu Select 'Element Solution', 'Stress', and 'X-Component of stress'. OK.

Element-Stresses

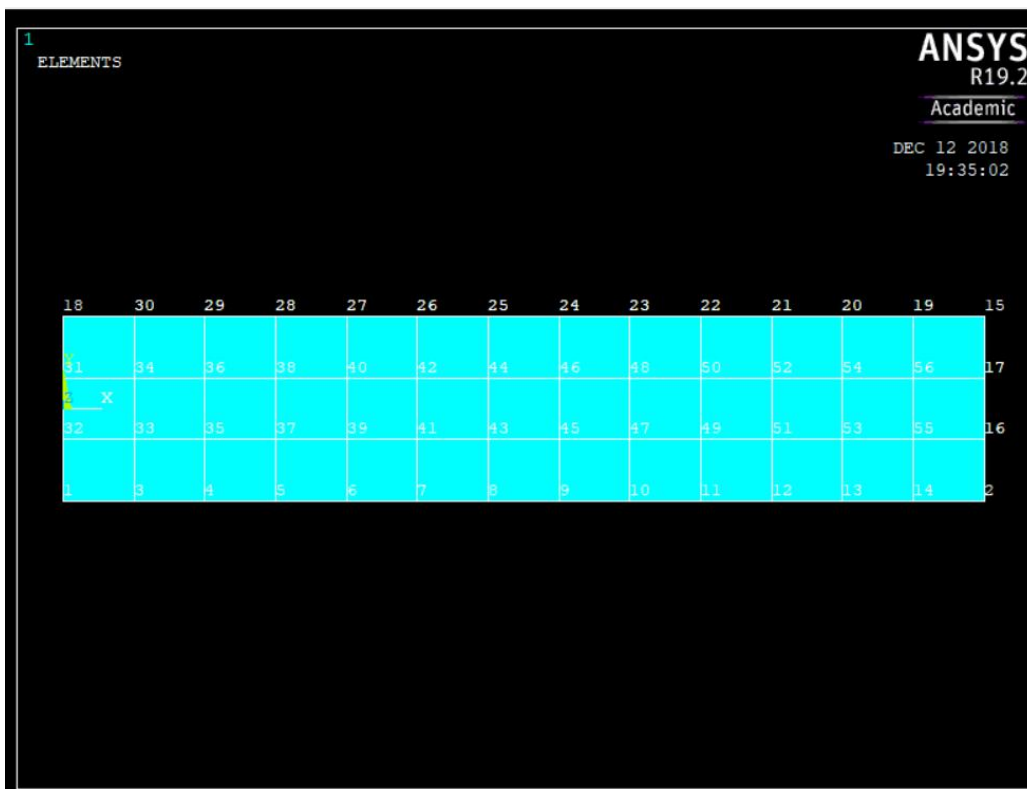


Nodal-Stresses



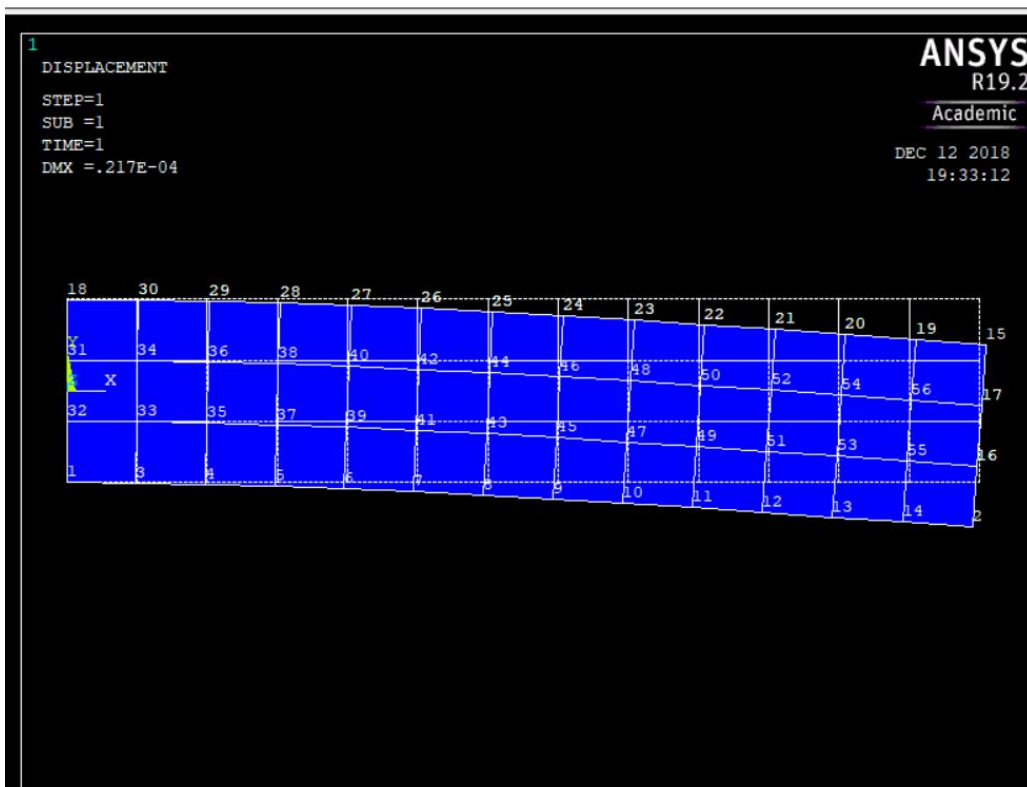
Ansys for fine Mesh:Follow the same procedure as of above

Mesh-size:0.5

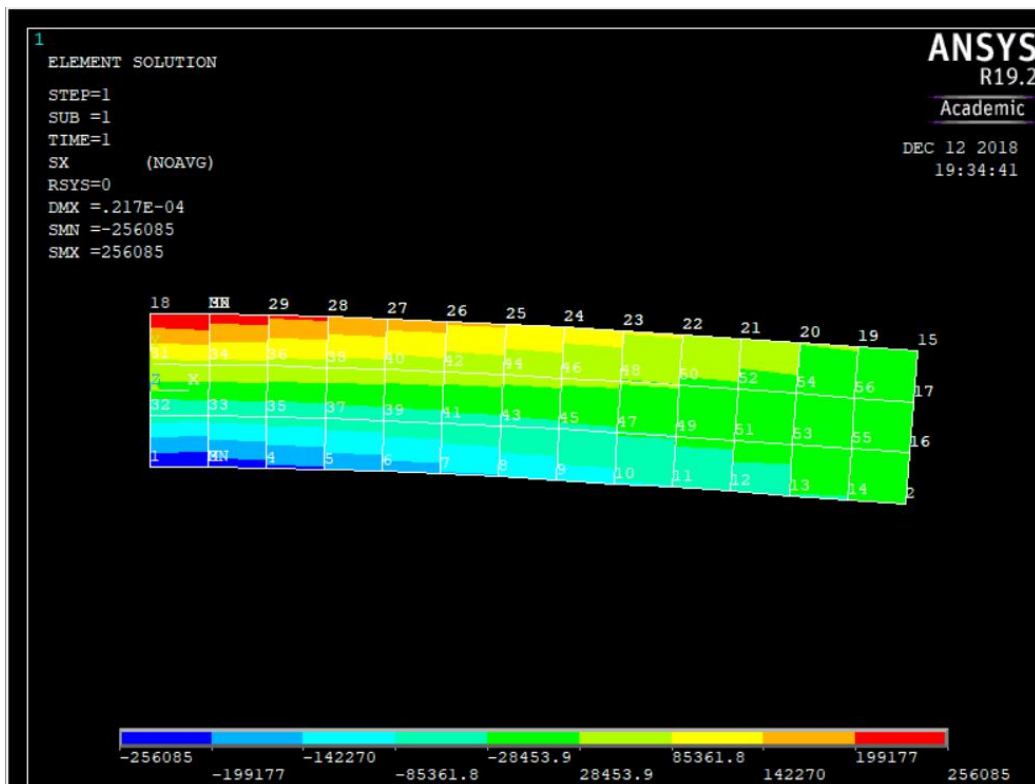


Nodal loads along y=node-15=-740.7,node 17=-3703.7,node 16=-3703.7,node-2=-740.7

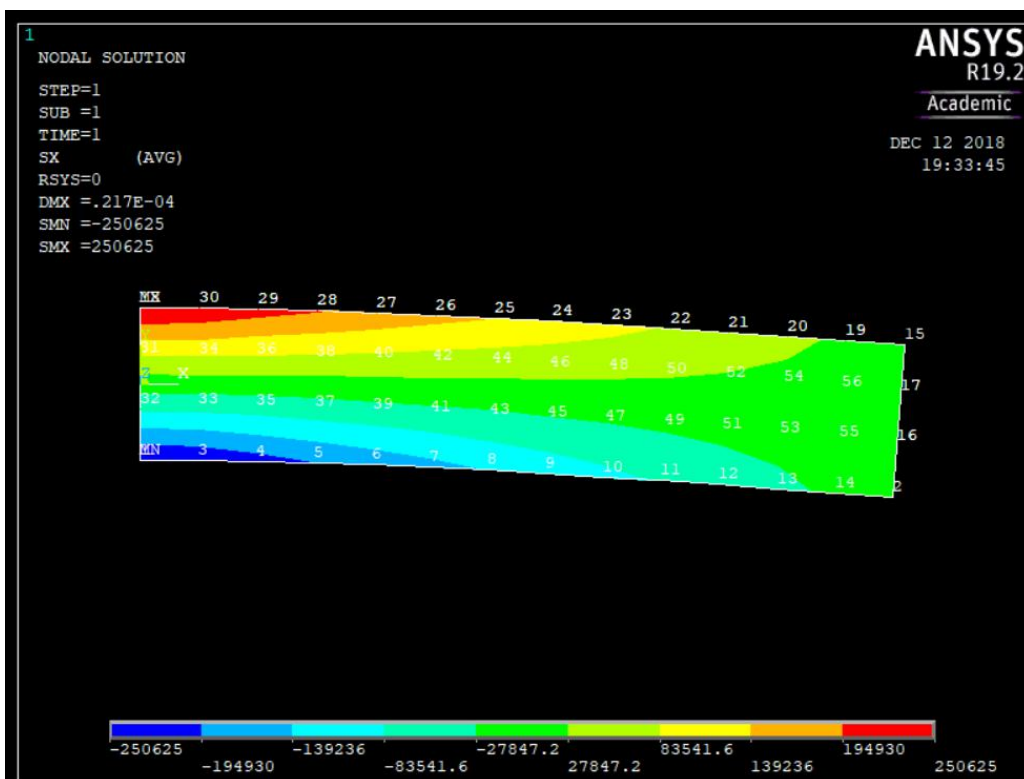
Deformed Shape



Element-Stress-X

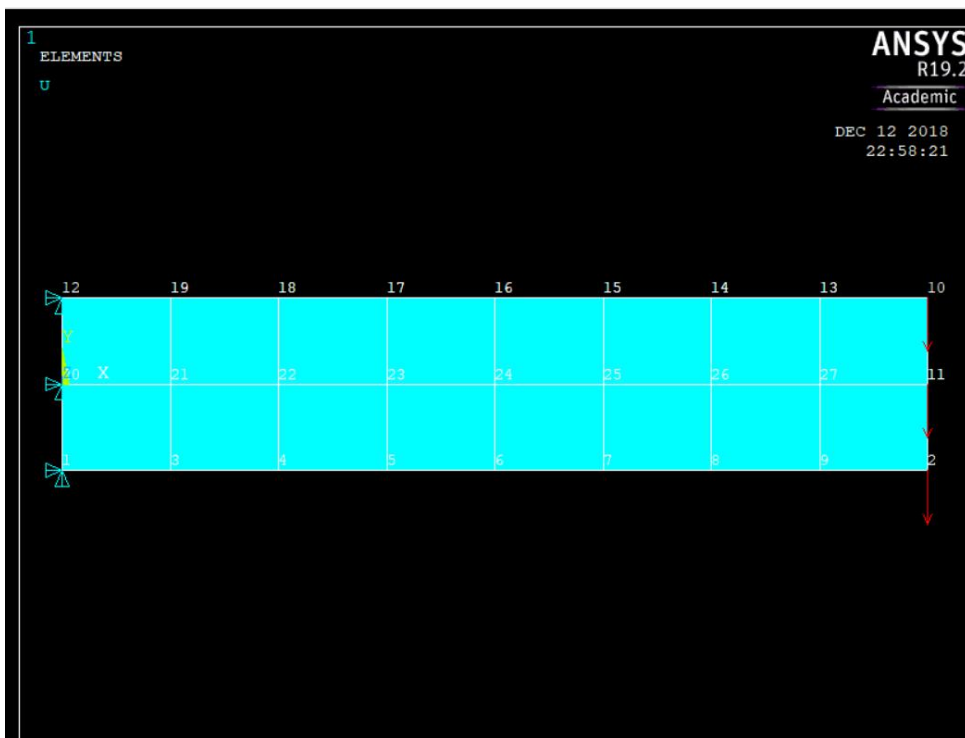


Nodal-Stress



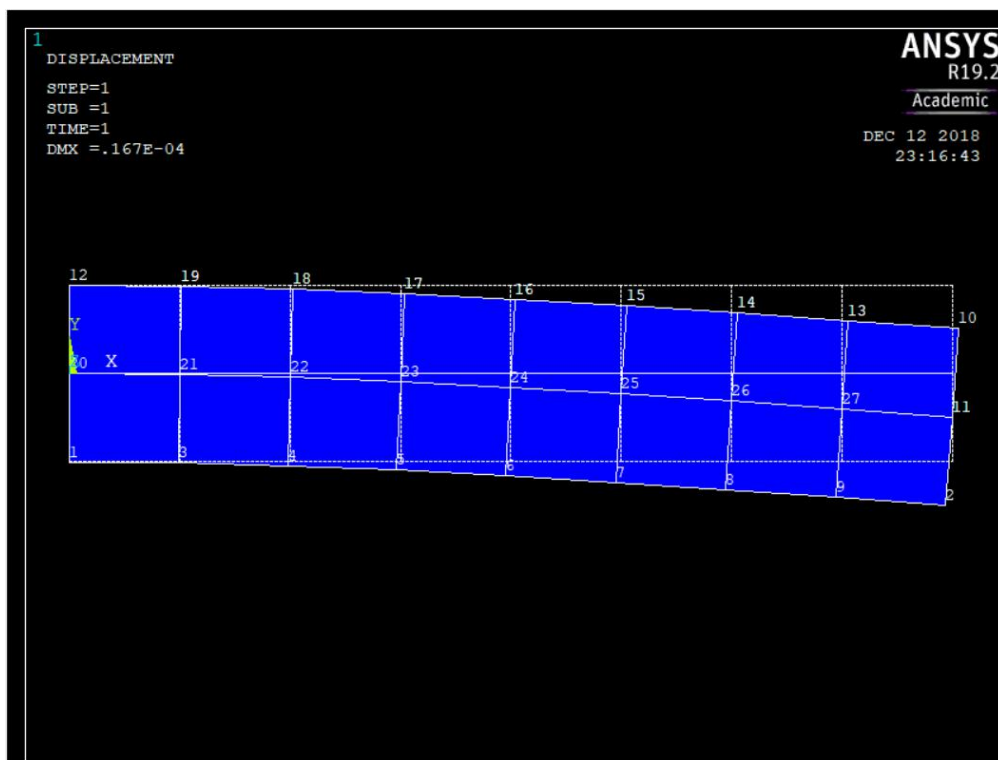
Ansys for coarse Mesh:-

Mesh-size-0.7

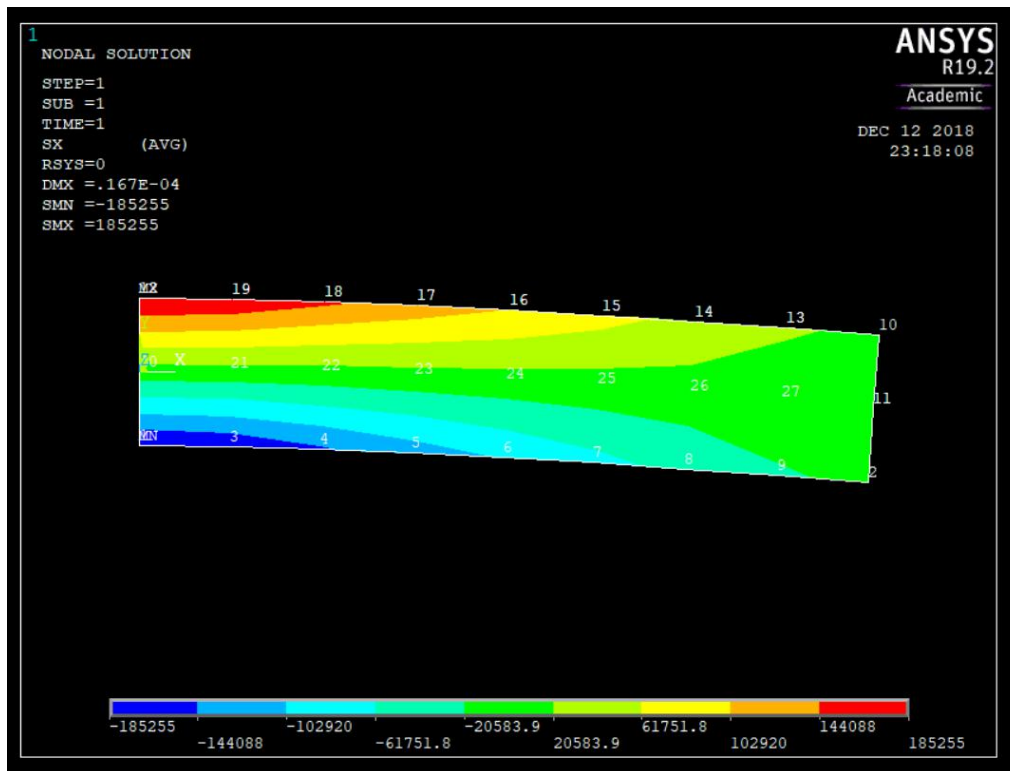


Deformed shape

Nodal-loads->node-10=-1250,node-11=-5000,node-2=-1250



Element-Stress



Observations from three types of mesh:-

- 1)As we make the mesh more fine then a very good convergence of the values can be seen.
- 2)As we decrease the mesh size the nodal loads at the end become more convergent to the applied load.

Ansys for Q8 element-Very fine mesh(0.3)

Preprocessing

Step 1. Define Element Type

ANSYS Main Menu: Preprocessor → Element Type → Add/Edit/Delete

In the pop-up window (Element Types), select Add. Then in the new window (Library of Element Types), select Structural Mass → Solid, Quad 8 node 183. OK to close.

In options, element behavior, pick 'plane stress'.

Step 2. Define Material Type

(1) ANSYS Main Menu: Preprocessor → Material Props → Material Models → Structural (double click) → Linear → Elastic → Isotropic

Input Mat'l Properties {EX=Young's '2. e11'; PRXY=0.3; OK}

Step3.Define Geometry

(1) Ansys Main Menu: Preprocessor → Modeling → Create → Areas → Rectangle → By 2 Corners
Enter X = 0; WP Y = -0.5; Width = 5; Height = 1, Click Apply to create rectangle.

Step 4. Create Mesh

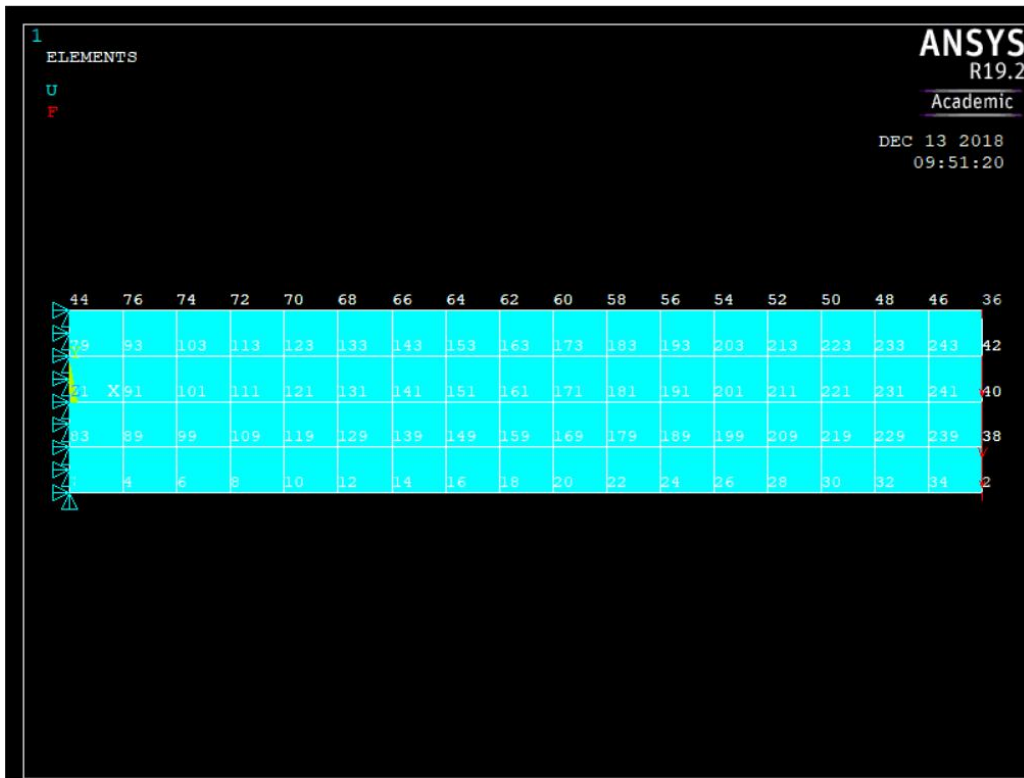
Ansyes Main Menu: Preprocessor → Meshing → Mesh Tool

A new "Mesh Tool" Window opens. Select Global.

In Size Controls panel, set global element SIZE as 0.3.

In Mesh panel, select Area, click Mesh. And pick the plate area to mesh

Mesh



Solving

Step 6. Apply Displacement Constraints:

Solve->Define Loads->Apply->Functions>

First define the displacement equations of UX, UY given and save the file.

Then Read the files UY.fun, UY.fun and names as DisplX and DisplY correspondingly.

Ansys Main Menu: Solution → Define Loads → Apply → Structural → Displacement → On nodes

Pick the left edge, select UX->instead of 'Constant value' use 'Existing Table' ->then select 'DisplX'. Repeat the same for UY also.

Step 7. Apply Loads

For Loads first find the individual nodal in MATLAB by using sigma-xy and shape functions

Ansys Main Menu: Solution → Define Loads → Apply → Structural → Force/Moment → On nodes

->pick->top node and give $F_y = -468.8$, similarly for bottom node-2 $F_y = -468.8$. node-42=-2500, node-40=-3437.5, Node-38=-2500

Step 8. Solve

Ansys Main Menu: Solution → Solve → Current LS

Close Solution Done Window.

Postprocessing

Step 9. Plot Deformed Shape

ANSYS Main Menu: General Postproc → Plot Results → Deformed Shape

Deformed Shape

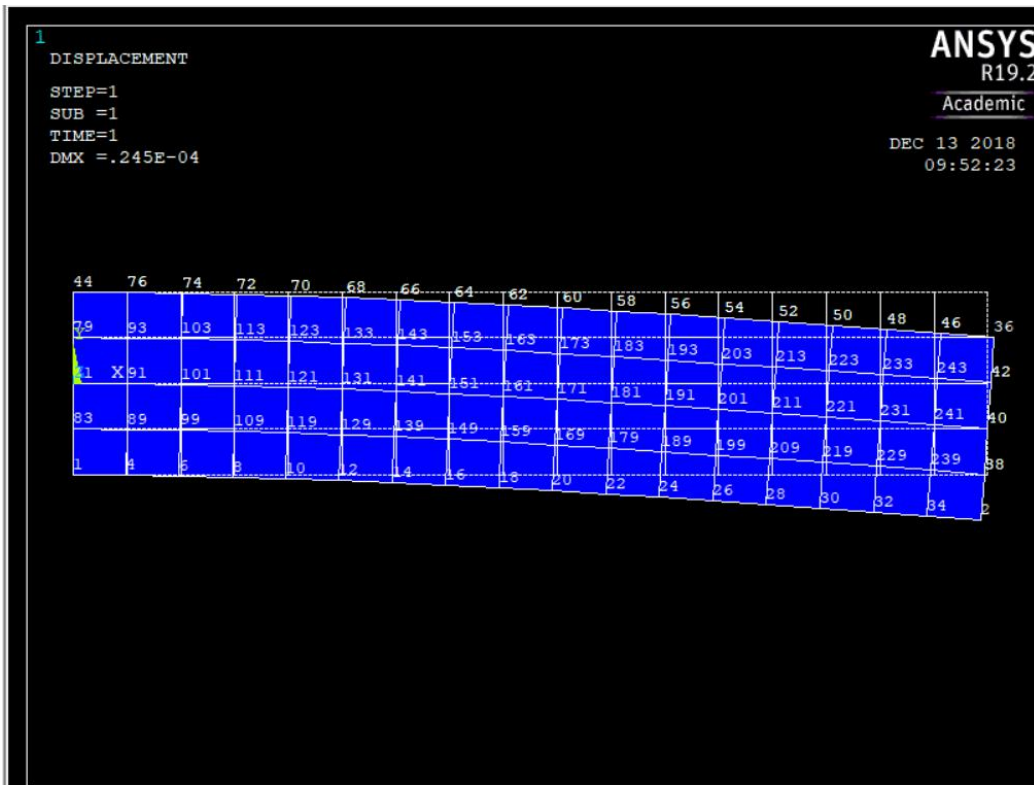
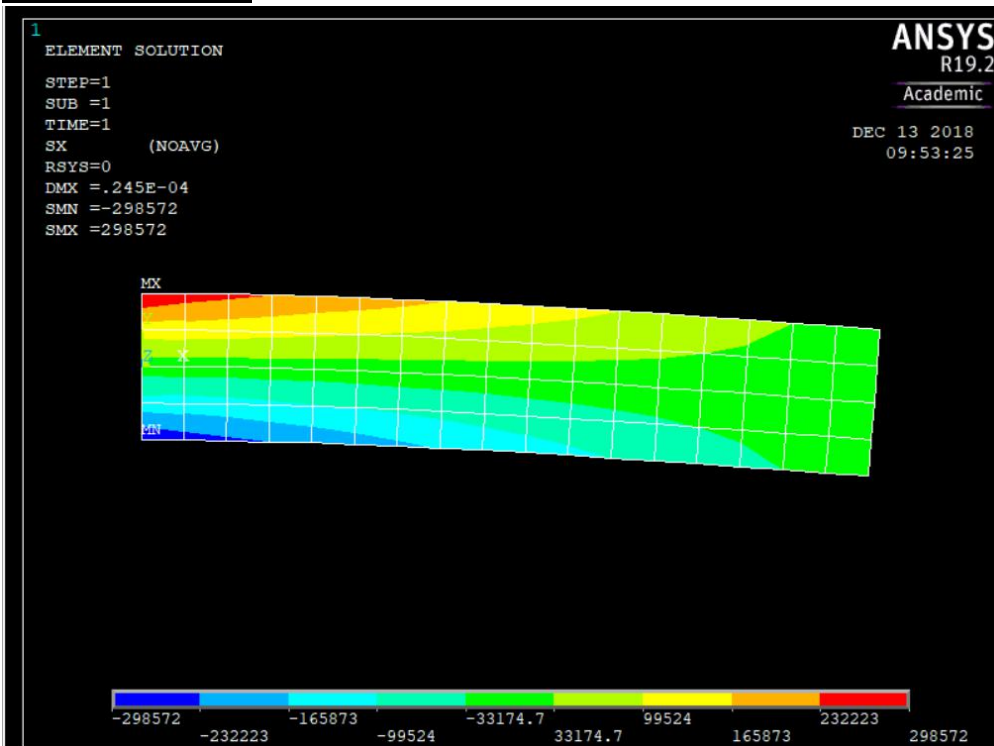


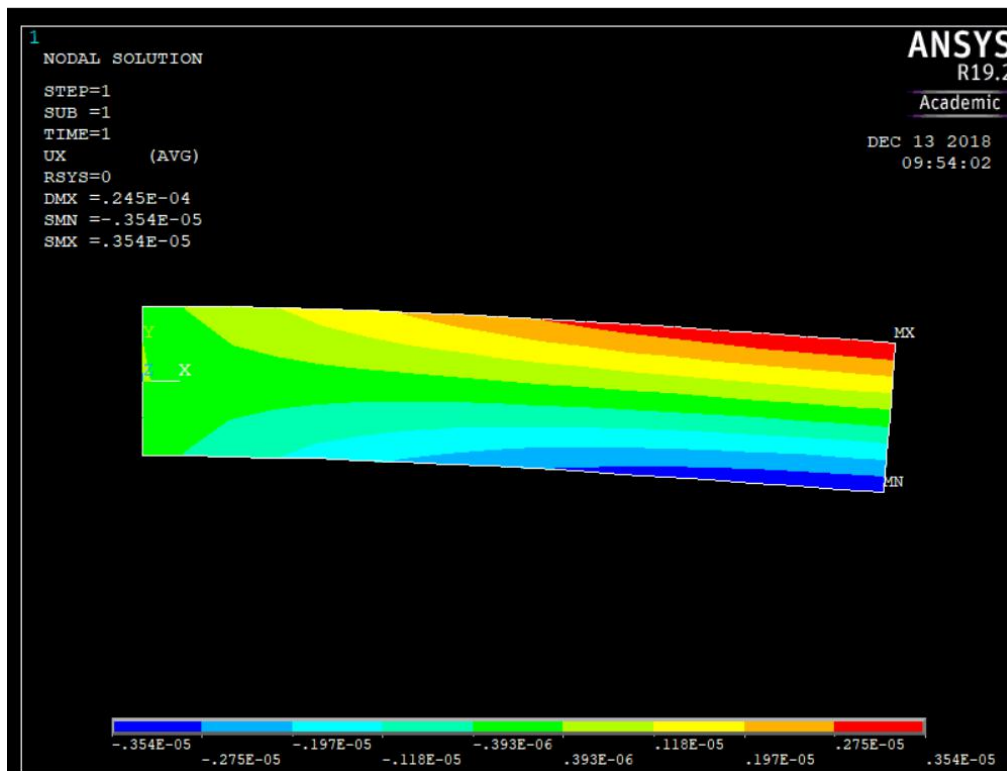
Fig.3 Deformed shape overlaid on outline of undeformed shape

Step 10. Contour Plots of Stresses ANSYS Main Menu: General Postproc → Plot Results → Contour Plot →Element Solu Select 'Element Solution', 'Stress', and 'X-Component of stress'. OK.

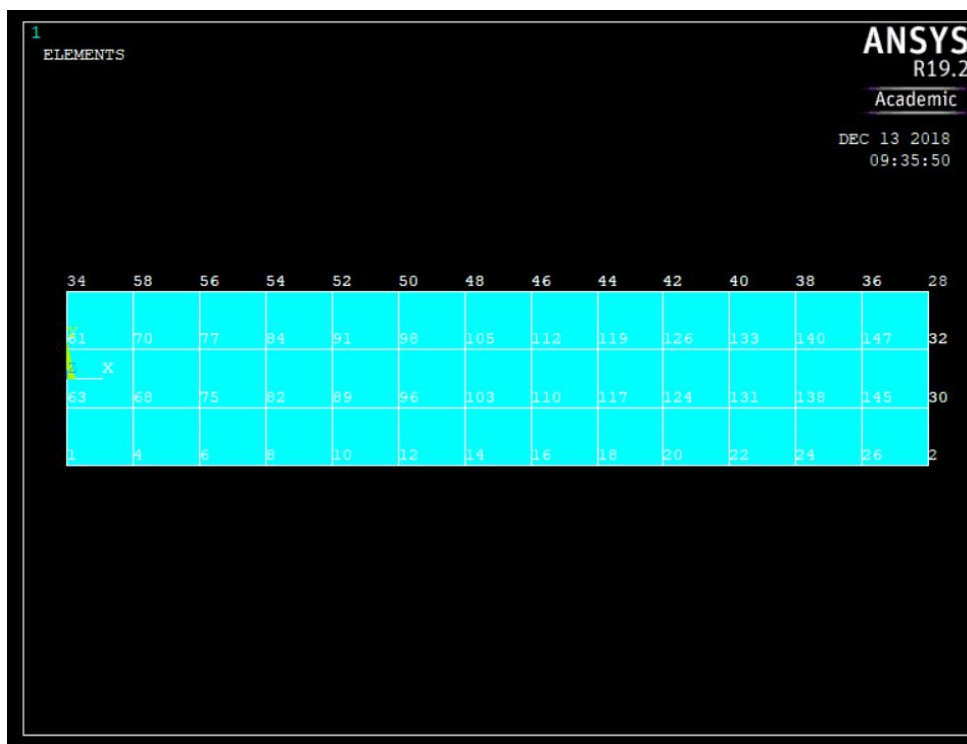
Element-Stress:-



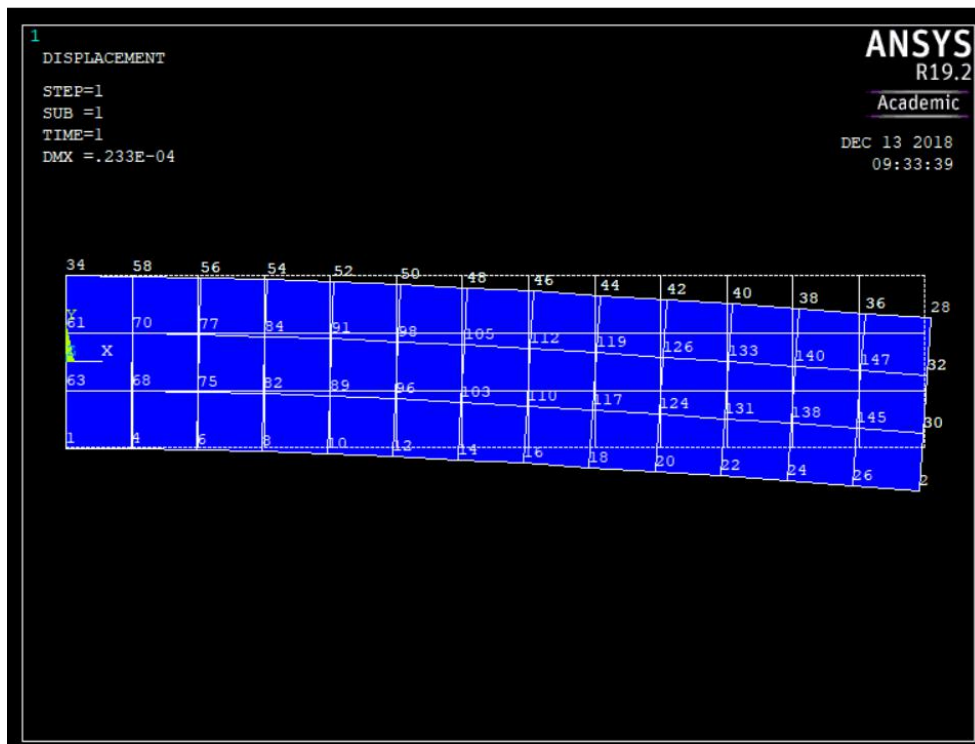
Displacement:



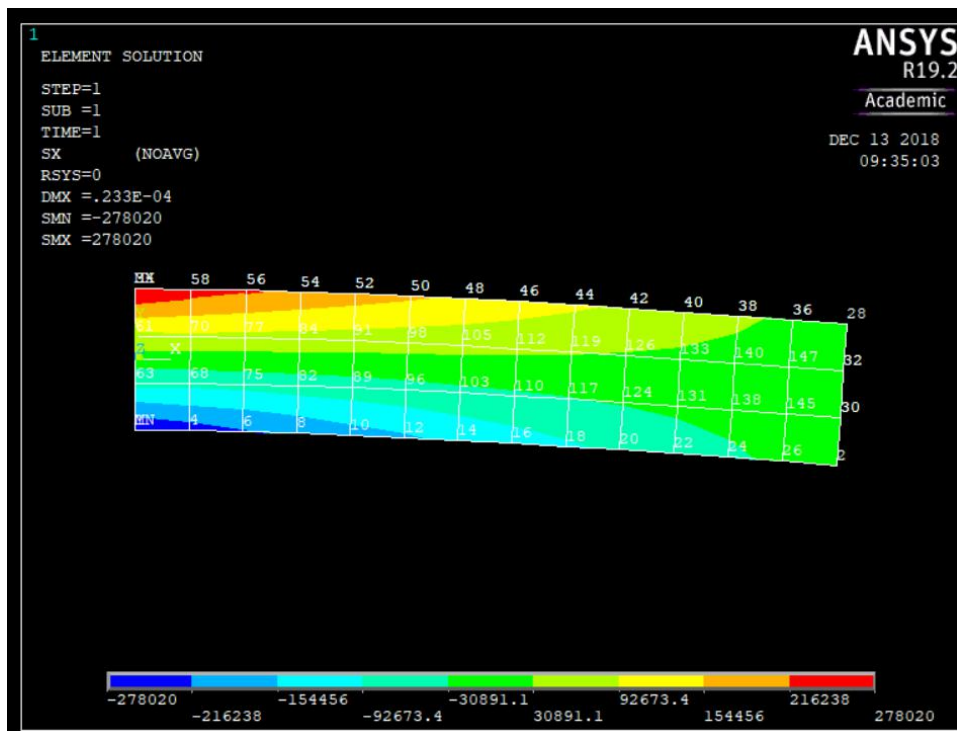
Similarly for Fine Mesh-



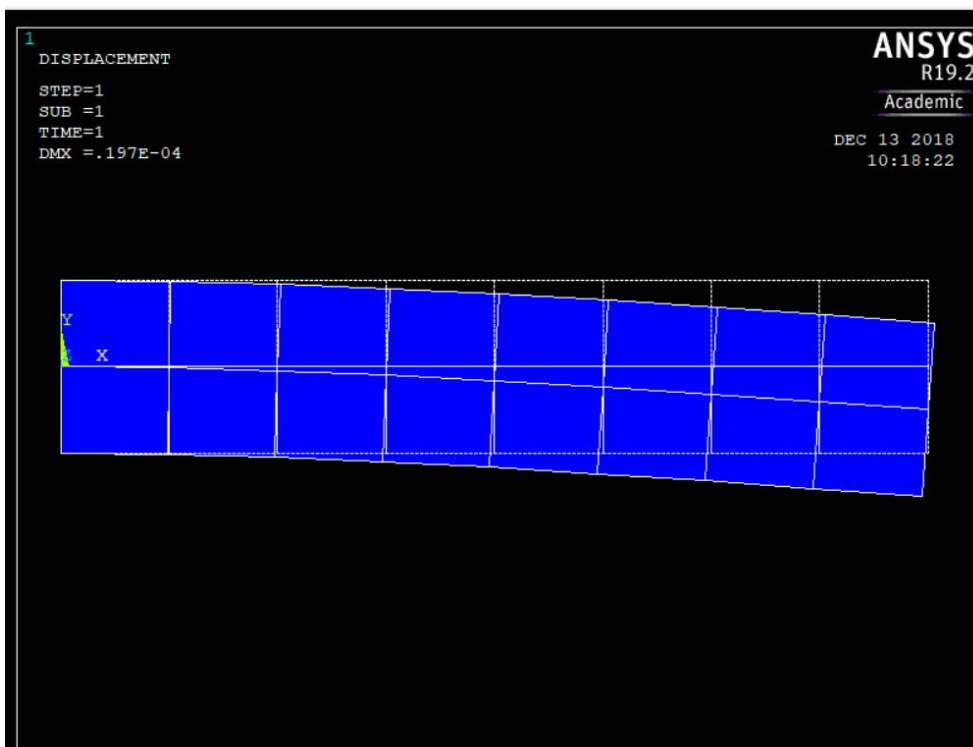
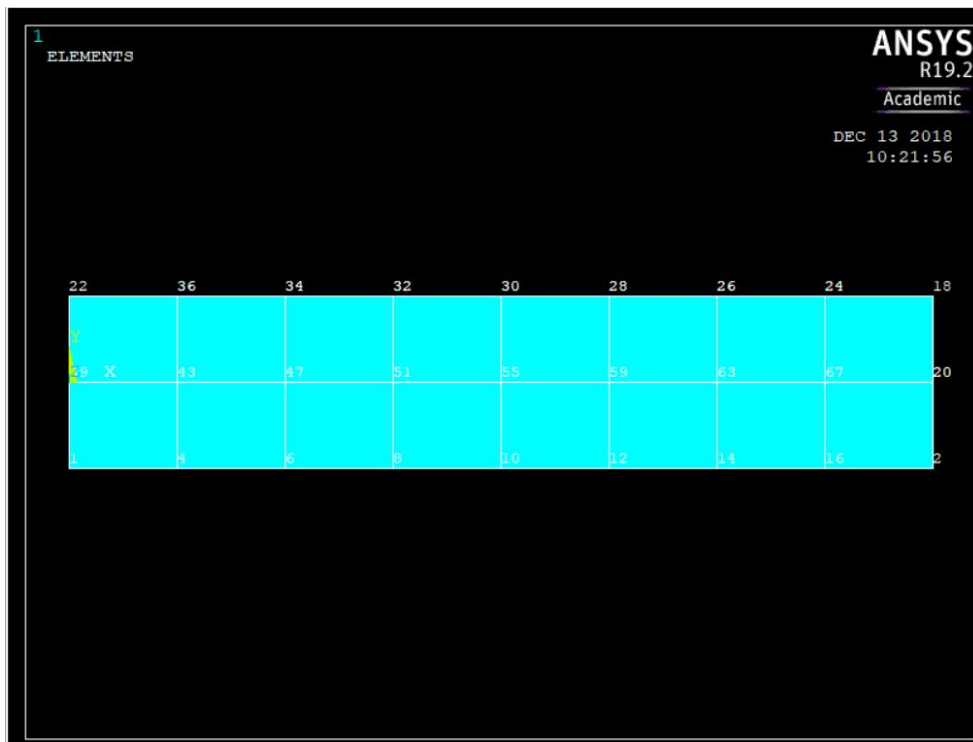
Deformed shape:-



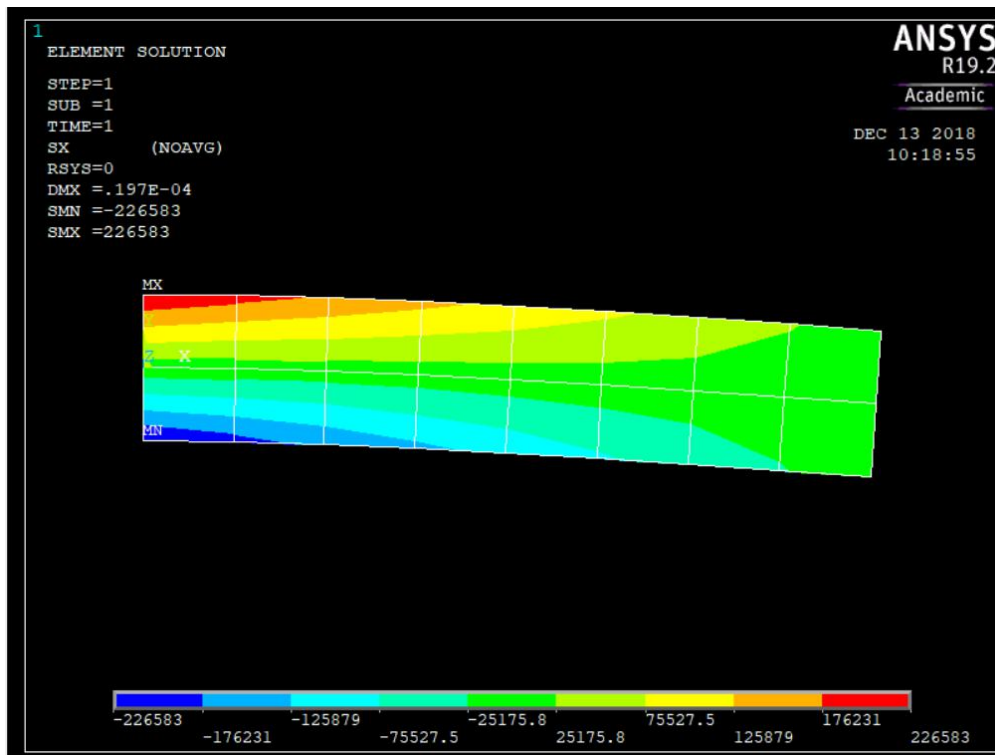
Stress-x



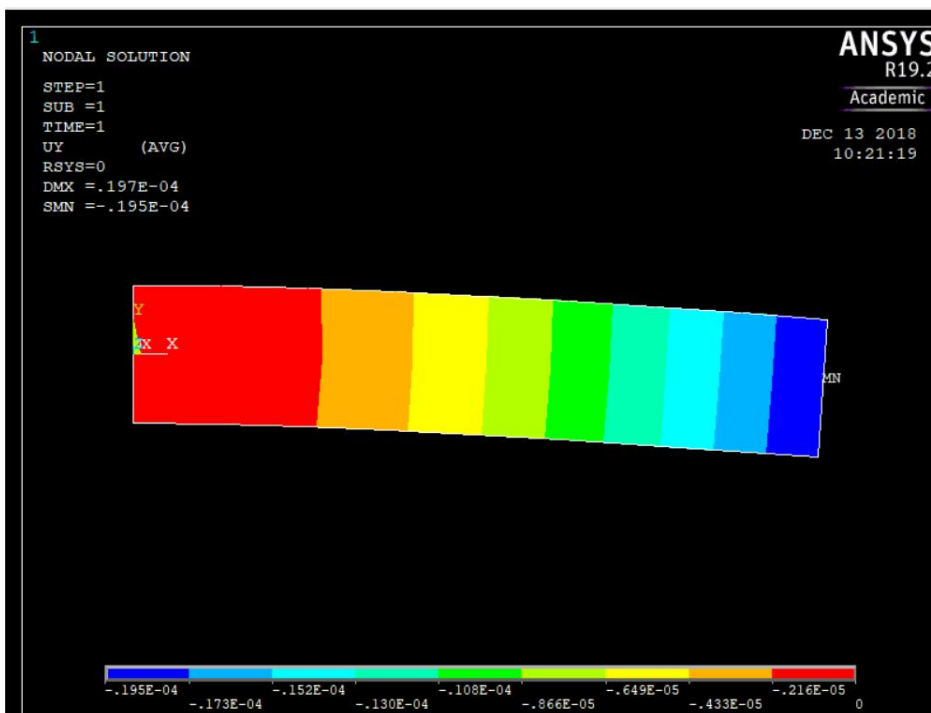
Q8 Coarse Mesh-(0.7)



Stress-X



Displacement



Sigma-x comparisons for different meshes at $x=L/2$

Considering nodes at $x=L/2$

Stress at $x=L/2$ -fine Mesh				Stress at $x=L/2$ -very fine Mesh				Stress at $x=L/2$ -very fine			
Nodes at $x=L/2$				nodes at $x=L/2$				Nodes at $x=L/2$			
46	1.23E+05			62	1.49E+05			18	-1.00E+05		
113	9.31E+04			164	1.29E+05			19	-82641		
112	41025			163	74449			20	-3.15E-05		
111	-21025			162	24563			21	82641		
110	-41025			161	1.33E-08			2	1.00E+05		
103	-9.31E+04			160	-24563						
1	-1.03E+05			159	-74449						
				158	-1.22E+05						
				20	-1.32E+05						

Observations:

- 1) As mesh size increases the values are more convergent to analytical results.
- 2) The stress values for corresponding nodes on both the sides are almost same but with different signs

2)Comparison of Ansys and Matlab Results for **Very Fine Mesh**(90 nodes) of Q4 element

Ansys-Displacement-Y

NODE	UY							
1	-1.13E-07							
2	-2.33E-05							
3	-2.53E-07							
4	-6.30E-07							
5	-1.21E-06							
6	-1.99E-06							
7	-2.94E-06							
8	-4.06E-06							
9	-5.33E-06							
10	-6.74E-06							
11	-8.27E-06							
12	-9.91E-06							
13	-1.16E-05							
14	-1.35E-05							
15	-1.54E-05							
16	-1.73E-05							
17	-1.93E-05							
18	-2.13E-05							
19	-2.33E-05							
20	-2.33E-05							
21	-2.33E-05							
22	-2.33E-05							
23	-1.13E-07							
24	-2.13E-05							
25	-1.93E-05							
26	-1.73E-05							
27	-1.54E-05							
28	-1.35E-05							
29	-1.16E-05							
30	-9.91E-06							
31	-8.27E-06							
32	-6.74E-06							
33	-5.33E-06							
34	-4.06E-06							

Matlab-Displacement Y

	-1.13E-07							
	-2.36E-05							
	-2.23E-07							
	-6.25E-07							
	-1.23E-06							
	-2.02E-06							
	-2.99E-06							
	-4.12E-06							
	-5.41E-06							
	-6.83E-06							
	-8.38E-06							
	-1.00E-05							
	-1.18E-05							
	-1.36E-05							
	-1.55E-05							
	-1.75E-05							
	-1.95E-05							
	-2.15E-05							
	-2.36E-05							
	-2.36E-05							
	-2.36E-05							
	-1.12E-07							
	-2.15E-05							
	-1.95E-05							
	-1.75E-05							
	-1.55E-05							
	-1.36E-05							
	-1.18E-05							
	-1.00E-05							
	-8.38E-06							
	-6.83E-06							
	-5.41E-06							
	-4.12E-06							

Maximum Value

34	-4.06E-06				-4.12E-06
35	-2.94E-06				-2.99E-06
36	-1.99E-06				-2.02E-06
37	-1.21E-06				-1.23E-06
38	-6.30E-07				-6.30E-07
39	-2.53E-07				-2.37E-07
40	-2.81E-08				1.25E-10
41	0				0
42	-2.81E-08				2.61E-10
43	-1.84E-07				-1.52E-07
44	-1.63E-07				-1.47E-07
45	-1.84E-07				-1.74E-07
46	-5.63E-07				-5.59E-07
47	-5.41E-07				-5.37E-07
48	-5.63E-07				-5.61E-07
49	-1.15E-06				-1.16E-06
50	-1.13E-06				-1.14E-06
51	-1.15E-06				-1.16E-06
52	-1.93E-06				-1.96E-06
53	-1.91E-06				-1.94E-06
54	-1.93E-06				-1.96E-06
55	-2.89E-06				-2.93E-06
56	-2.87E-06				-2.91E-06
57	-2.89E-06				-2.93E-06
58	-4.01E-06				-4.07E-06
59	-3.99E-06				-4.06E-06
60	-4.01E-06				-4.07E-06
61	-5.28E-06				-5.36E-06
62	-5.27E-06				-5.35E-06
63	-5.28E-06				-5.36E-06
64	-6.69E-06				-6.79E-06
65	-6.68E-06				-6.78E-06
66	-6.69E-06				-6.79E-06
67	-8.23E-06				-8.34E-06
68	-8.22E-06				-8.33E-06
69	-8.23E-06				-8.34E-06
70	-9.88E-06				-1.00E-05

33	1.63E+05	13118	-4507.2		Columns 61 through 72														
34	1.79E+05	14430	-4507.3																
35	1.96E+05	15743	-4506.8	SX	-0.7914	-0.0393	0.7127	-0.7082	-0.0354	0.6373	-0.6244	-0.0315	0.5614	-0.5409	-0.0275	0.4858			
36	2.12E+05	17044	-4505.7	SY	-0.1181	-0.1312	-0.1443	-0.1050	-0.1181	-0.1311	-0.0918	-0.1049	-0.1181	-0.0787	-0.0918	-0.1049			
37	2.28E+05	18330	-4510.4	SXY	-0.2967	-0.3005	-0.2368	-0.2801	-0.2840	-0.2202	-0.2606	-0.2645	-0.2008	-0.2426	-0.2465	-0.1828			
38	2.45E+05	19954	-4726.1																
39	2.67E+05	25062	-4149.7		Columns 73 through 84														
40	1.22E+05	-8512.6	-40672																
41	-7.19E-05	4.11E-04	-39226	SX	-0.4574	-0.0236	0.4102	-0.3739	-0.0197	0.3345	-0.2903	-0.0157	0.2588	-0.2070	-0.0118	0.1833			
42	-1.22E+05	8512.6	-40672	SY	-0.0656	-0.0787	-0.0918	-0.0525	-0.0656	-0.0787	-0.0393	-0.0525	-0.0656	-0.0263	-0.0393	-0.0525			
43	-1.23E+05	-985.81	-8394.6	SXY	-0.2246	-0.2285	-0.1648	-0.2066	-0.2105	-0.1468	-0.1886	-0.1926	-0.1288	-0.1725	-0.1762	-0.1126			
44	-0.36633E-004-	4.07E-05	-11150																
45	1.23E+05	985.81	-8394.6		Columns 85 through 90														
46	-1.18E+05	144.38	-8603.1																
47	-1.12E-05	4.42E-05	-11665	SX	-0.1249	-0.0080	0.1092	-0.0392	-0.0042	0.0309									
48	1.18E+05	-144.38	-8603.1	SY	-0.0132	-0.0267	-0.0388	0.0009	-0.0139	-0.0284									
49	-1.11E+05	18.504	-8677.4	SXY	-0.1523	-0.1561	-0.0925	-0.1340	-0.1404	-0.0761									
50	-3.57E-05	2.07E-04	-12050																
51	1.11E+05	-18.504	-8677.4																
52	-1.03E+05	-1.5695	-8670.2																
53	-5.41E-05	6.39E-05	-12059																
54	1.03E+05	1.5691	-8670.2																
55	-95498. -	0.35601	-8670.1																
56	-2.39E-05	8.32E-05	-12052																
57	95498	0.35634	-8670.1																
58	-87540	1.98E-02	-8669.9																
59	-0.50137E-004-	4.34E-06	-12052																
60	-87540	1.97E-02	-8669.9																
61	-79582	5.15E-03	-8670																
62	2.66E-05	8.86E-05	-12052																
63	-79582	5.20E-03	-8670																
64	-71624	9.47E-05	-8670																
65	-8.92E-06	1.09E-05	-12052																
66	-71624	1.65E-04	-8670																
67	-63665. -	3.60E-04	-8670																

67	-63665. -	3.60E-04	-8670
68	-0.13377E-004-	8.53E-05	-12052
69	63665	2.71E-04	-8670
70	-55707	2.17E-06	-8670
71	-0.13418E-004-	8.54E-05	-12052
72	55707	2.16E-04	-8670
73	-47749. -	7.37E-04	-8670
74	-0.28827E-004-	5.54E-05	-12052
75	47749	8.34E-04	-8670
76	-39791	3.78E-02	-8670
77	-2.49E-06	3.24E-05	-12052
78	-39791	3.78E-02	-8670
79	-31831. -	1.54E-02	-8669.8
80	1.50E-05	5.00E-05	-12053
81	31831	1.55E-02	-8669.8
82	-23877	-1.8416	-8671.2
83	-8.69E-06	1.17E-05	-12049
84	23877	1.8415	-8671.2
85	-15997	-5.4264	-8659
86	1.27E-06	1.44E-05	-12027
87	15997	5.4264	-8659
88	-7980.9	125.08	-8687.4
89	-1.36E-06	5.38E-07	-12092
90	7980.9	-125.08	-8687.4

MINIMUM VALUES

NODE 1 3 40
VALUE -0.27649E+006 - 25062. -40672

MAXIMUM VALUES

NODE 23 39 3
VALUE 2.76E+05 25062. -4149.7

Comparison of Ansys and Matlab Results for **Fine-Mesh** of Q4 element

Ansys-Y-Displacement			Matlab Y-Displacement		
NODE	UY				
1	-1.13E-07		-1.13E-07		
2	-2.15E-05	Max	-2.13E-05	Max	
3	-3.21E-07		-3.49E-07		
4	-8.92E-07		-8.80E-07		
5	-1.77E-06		-1.75E-06		
6	-2.94E-06		-2.90E-06		
7	-4.36E-06		-4.30E-06		
8	-6.00E-06		-5.94E-06		
9	-7.85E-06		-7.77E-06		
10	-9.87E-06		-9.77E-06		
11	-1.20E-05		-1.19E-05		
12	-1.43E-05		-1.42E-05		
13	-1.67E-05		-1.65E-05		
14	-1.91E-05		-1.89E-05		
15	-2.15E-05		-2.13E-05		
16	-2.15E-05		-2.13E-05		
17	-2.15E-05		-2.13E-05		
18	-1.13E-07		-1.13E-07		
19	-1.91E-05		-1.89E-05		
20	-1.67E-05		-1.65E-05		
21	-1.43E-05		-1.42E-05		
22	-1.20E-05		-1.19E-05		
23	-9.87E-06		-9.77E-06		
24	-7.85E-06		-7.77E-06		
25	-6.00E-06		-5.94E-06		
26	-4.36E-06		-4.30E-06		
27	-2.94E-06		-2.90E-06		
28	-1.77E-06		-1.75E-06		
29	-8.92E-07		-8.87E-07		
30	-3.21E-07		-3.27E-07		
31	-1.25E-08		-8.61E-09		
32	-1.25E-08		-1.13E-07		
33	-2.48E-07		-2.38E-07		
34	-2.48E-07		-2.61E-07		
35	-8.21E-07		-8.15E-07		
36	-8.21E-07		-8.12E-07		
37	-1.71E-06		-1.68E-06		
38	-1.71E-06		-1.69E-06		
39	-2.88E-06		-2.84E-06		
40	-2.88E-06		-2.84E-06		
41	-4.31E-06		-4.25E-06		
42	-4.31E-06		-4.25E-06		
43	-5.96E-06		-5.89E-06		
44	-5.96E-06		-5.89E-06		
45	-7.81E-06		-7.73E-06		
46	-7.81E-06		-7.73E-06		
47	-9.84E-06		-9.74E-06		
48	-9.84E-06		-9.74E-06		
49	-1.20E-05		-1.19E-05		
50	-1.20E-05		-1.19E-05		
51	-1.43E-05		-1.42E-05		
52	-1.43E-05		-1.42E-05		
53	-1.66E-05		-1.65E-05		
54	-1.66E-05		-1.65E-05		
55	-1.91E-05		-1.89E-05		
56	-1.91E-05		-1.89E-05		
MAXIMUM	ABSOLUTE VALUES				
NODE	16				
VALUE	-2.15E-05				

Stress-X values Comparision at nodes:-

Ansys Stress Results					MATLAB RESULTS FOR STRESSES AT NODES																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																									
	NODE	SX	SY	SXY		NODES 1	TO	NODES 12																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																						

39	-57621	2.2853	-8712
40	57621	-2.2854	-8712
41	-51224	0.13827	-8713.3
42	51224	0.13887	-8713.3
43	-44821	0.69305E	-8713.2
44	-44821	0.66073E	-8713.2
45	-38418	-0.10975E	-8713.2
46	38418	0.75021E	-8713.2
47	-32015	0.13904E	-8713.2
48	-32015	0.13586E	-8713.2
49	-25613	0.23881	-8713.1
50	25613	0.23842	-8713.1
51	-19201	3.3454	-8714.9
52	19201	-3.3454	-8714.9
53	-12876	-35.145	-8695.9
54	12876	35.145	-8695.9
55	-6417.6	153.83	-8763.6
56	6417.6	-153.83	-8763.6
MINIMUM VALUES			
NODE	1	3	31
VALUE	-0.25062E+	29253	-46148
MAXIMUM VALUES			
NODE	18	30	3
VALUE	2.51E+05	29253	-3879.5

Comparisons between Ansys and Matlab for Coarse Mesh(27 Nodes)

A	B	C	D	E	F	G	H
Ansys Displacement-Y					MATLAB DISPLACEMENT-Y		
NODE	UY						
1	-1.13E-07				-1.13E-07		
2	-1.65E-05				-1.69E-05		
3	-5.00E-07				-5.52E-07		
4	-1.58E-06				-1.67E-06		
5	-3.21E-06				-3.36E-06		
6	-5.30E-06				-5.52E-06		
7	-7.78E-06				-8.05E-06		
8	-1.05E-05				-1.09E-05		
9	-1.35E-05				-1.39E-05		
10	-1.65E-05				-1.69E-05		
11	-1.65E-05				-1.68E-05		
12	-1.13E-07				-1.13E-07		
13	-1.35E-05				-1.39E-05		
14	-1.05E-05				-1.09E-05		
15	-7.78E-06				-8.05E-06		
16	-5.30E-06				-5.52E-06		
17	-3.21E-06				-3.36E-06		
18	-1.58E-06				-1.67E-06		
19	-5.00E-07				-5.31E-07		
20	0				2.15E-18		
21	-4.46E-07				-4.75E-07		
22	-1.52E-06				-1.61E-06		
23	-3.16E-06				-3.31E-06		
24	-5.27E-06				-5.48E-06		
25	-7.75E-06				-8.02E-06		
26	-1.05E-05				-1.08E-05		
27	-1.35E-05				-1.38E-05		
MAXIMUM ABSOLUTE VALUES							
NODE	11						
VALUE -	1.65E-05						
VALUE -	0.16505E-004						

Ansys_Stress at nodes								MATLAB STRESS AT NODES												
	NODE	SX	SY	SZ	SXY	SYZ	SXZ													
Maximum	1	-1.85E+05	-10559	0	-47696	0	0	The Nodal Stresses are												
	2	-13428	-5783.7	0	-3811.3	0	0	Columns 1 through 12												
	3	-1.78E+05	-31640	0	-2499.1	0	0	1.0E+05 *	-1.8519	-0.1343	-1.5950	-1.3801	-1.1246	-0.8733	-0.6219	-0.3718	-0.1155	0.1343		
	4	-1.50E+05	-22846	0	-4394.1	0	0		-0.5556	-0.0578	-0.1728	-0.2010	-0.1582	-0.1184	-0.0787	-0.0405	0.0049	0.0578		
	5	-1.26E+05	-19898	0	-4003.7	0	0		-0.5057	-0.0381	-0.4123	-0.3896	-0.3286	-0.2708	-0.2132	-0.1566	-0.0943	-0.0381		
	6	-1.00E+05	-15751	0	-4085.8	0	0	Columns 13 through 24												
	7	-75356	-11826	0	-4060.6	0	0	SX	0.1155	0.3718	0.6219	0.8735	1.1242	1.3721	1.6548	-0.1981	-0.1034	-0.0656		
	8	-50290	-7979	0	-4106	0	0	SY	-0.0049	0.0405	0.0787	0.1187	0.1563	0.1991	0.2739	-0.5096	-0.2536	-0.2323		
	9	-24881	-3512.7	0	-3901.2	0	0	SXY	-0.0943	-0.1566	-0.2132	-0.2710	-0.3281	-0.3831	-0.4709	-0.5848	-0.4740	-0.3928		
	10	13428	5783.7	0	-3811.3	0	0	Columns 25 through 27												
	11	2.10E-06	2.74E-05	0	-5567.3	0	0	SX	-0.0356	-0.0234	-0.0130									
	12	1.85E+05	10559	0	-47696	0	0	SY	-0.1186	-0.0781	-0.0434									
	13	24881	3512.7	0	-3901.2	0	0	SXY	-0.2245	-0.1663	-0.1119									
	14	50290	7979	0	-4106	0	0													
	15	75356	11826	0	-4060.6	0	0													
	16	1.00E+05	15751	0	-4085.8	0	0													
	17	1.26E+05	19898	0	-4003.7	0	0													
	18	1.50E+05	22846	0	-4394.1	0	0													
	19	1.78E+05	31640	0	-2499.1	0	0													
	20	1.44E-04	-8.22E-04	0	-54846	0	0													
	21	3.36E-05	-1.32E-04	0	-6004.4	0	0													
	22	-1.38E-05	3.54E-05	0	-5004.7	0	0													
	23	5.68E-05	-2.17E-04	0	-5210.6	0	0													
	24	-3.15E-05	5.79E-05	0	-5167.4	0	0													
	25	-2.31E-05	4.26E-06	0	-5180.6	0	0													
	26	1.07E-05	-4.57E-05	0	-5156.7	0	0													
	27	-4.00E-06	-1.33E-05	0	-5264.7	0	0													
MINIMUM VALUES																				
NODE	1		3		1	20	1	1												
VALUE	-1.85E+05		-31640		0	-54846	0	0												
MAXIMUM VALUES																				
NODE	12		19		1	3	1	1												
VALUE	1.85E+05		31640		0	-2499.1	0	0												

3) Sigma-X at $x=L/2$ for three types of mesh:

The nodes which are at $x=L/2$ are Considered

Sigma-X at $x=L/2$:Very Fine Mesh

Ansys_STRESS-X at $x=L/2$		MATLAB STRESS-X at $X=L/2$	
Nodes at $X=L/2$			
32	1.47E+05	1.41E+05	
66	71624	70624	
65	8.92E-06	9.82E-06	
64	1.19E+05	1.19E+05	
10	-1.47E+05	-1.40E+05	

Sigma-X at $x=L/2$:Fine Mesh

ANSYS Sigma X at $x=L/2$		MATLAB STRESS-X at $X=L/2$ 2-Point	
Nodes at $x=L/2$			
9	-1.19E+05	-1.25E+05	
45	-38418	-45418	
46	38418	45418	
24	1.19E+05	1.25E+05	

Sigma-X at $x=L/2$:Coarse Mesh

Ansys Stress at $x=L/2$		Matlab Stress at $x=L/2$ ---2point Gauss	
Nodes at $x=L/2$			
6	-1.00E+05	-9.00E+04	
24	-3.15E-05	-2.90E+05	
16	1.00E+05	9.00E+04	

3)Comparison of Stress(sigma-x) at 2-point,1-point,3-point Gauss Quadrature

ANSYS Sigma X at $x=L/2$		MATLAB STRESS-X at $X=L/2$ 2-Point	MATLAB STRESS-X at $X=L/2$ 1-Point	MATLAB STRESS-X at $X=L/2$ 3-Point
Nodes at $x=L/2$				
9	-1.19E+05	-1.25E+05	3.10E+05	-1.09E+05
45	-38418	-45418	-68418	-38700
46	38418	45418	68418	38700
24	1.19E+05	1.25E+05	-309930.00	1.09E+05