

FINITE ELEMENT TECHNIQUES(FALL 2018)

Final Project

Damacharla,vishnu Vardhan

vxd180005@utdallas.edu

Objective: -Q4 and Q8 element implementation for a cantilever beam with end load in MATLAB for different mesh sizes under plane stress state and comparing stresses and deflection with Ansys results.

Implementation in MATLAB: -

1) Plane stress analysis of solid using linear Q4 elements

Variable descriptions

k = element matrix

f = element vector

kk = system matrix

ff = system vector

displ = system nodal displacement vector

eldisp = element nodal displacement vector

gcoord = coordinate values of each node

nodes = nodal connectivity of each element

index = a vector containing system dofs associated with each element

bcdof = a vector containing dofs associated with boundary conditions

bcval = a vector containing boundary condition values associated with the dofs in 'bcdof'

q=shear stress equation

ForceNodes=nodes on which load is acting

ShapFun=Shape functions matrix;

DiffMatr=Differentiation matrix

Jacobian=jacobian matrix

TransMatrx=Transformation matrix

Bmtx2='B' matrix

DisplX=Displacement along X,DisplY=Displacement along Y

NodeStrain=Nodal strains

NodeStress=nodal stresses

ElemeStrains=Element strains

ElemeStresses=Element stresses

eleStrain.m file consists of element strain calculation.

Input data:-D=1; Load=10000, E=2e11, Length=5, Poisson ratio=0.3

Procedure: -

Meshing with Element sizes of 0.3,0.4,0.7

1) Initializing all the input variables like number of elements, number of nodes, number of degrees of freedom etc.

2) Importing the nodal coordinates, mesh data and force nodes from Ansys through excel File named 'Data.xlsx'. Then we have find the loads on the nodes using the shear stress and shape functions matrix.

3)Ux, Uy equations are used for specifying the displacement boundary conditions.

4) Calculating a Constitutive matrix for plain stress analysis.

Here in command window you are prompted to enter Desired gaussian quadrature points 1,2 or 3

5)finding Jacobean, transformation matrix, differentiation matrix and B matrix

6) Then Calculating element stiffness matrix using 2X2 ,3,1-point gaussian quadrature;

7) Assembling of the elements.

8) Finding the system matrix.

9) Then finding out stress at different nodes and elements.

Anslys for very fine Mesh-Q4 element

Preprocessing

Step 1. Define Element Type

ANSYS Main Menu: Preprocessor → Element Type → Add/Edit/Delete

In the pop-up window (Element Types), select Add. Then in the new window (Library of Element Types), select Structural Mass → Solid, Quad 4 node 182. OK to close.

In options, element behavior, pick 'plane stress'.

Step 2. Define Material Type

(1) ANSYS Main Menu: Preprocessor → Material Props → Material Models → Structural (double click) → Linear → Elastic → Isotropic

Input Mat'l Properties {EX=Young's '2. e11'; PRXY=0.3; OK}

Step3. Define Geometry

(1) Ansys Main Menu: Preprocessor → Modeling → Create → Areas → Rectangle → By 2 Corners
Enter X = 0; WP Y = -0.5; Width = 5; Height = 1, Click Apply to create rectangle.

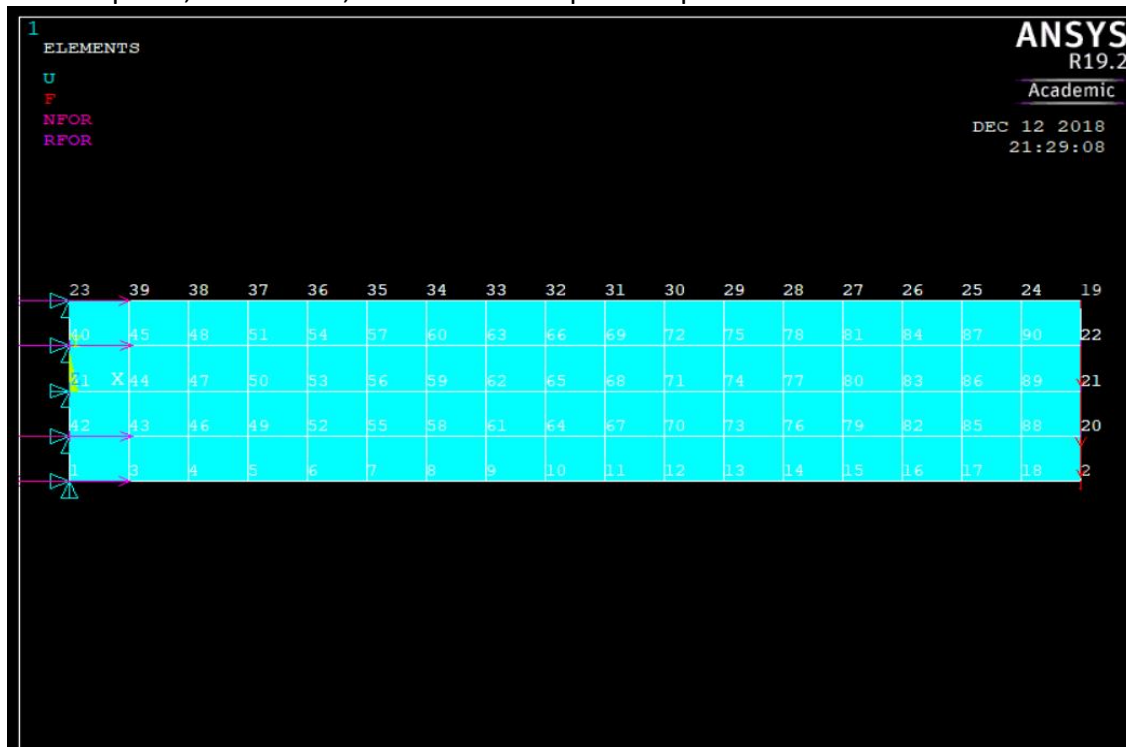
Step 4. Create Mesh

Anslys Main Menu: Preprocessor → Meshing → Mesh Tool

A new "Mesh Tool" Window opens. Select Global.

In Size Controls panel, set global element SIZE as 0.3.

In Mesh panel, select Area, click Mesh. And pick the plate area to mesh



Solving

Step 6. Apply Displacement Constraints:

Solve->Define Loads->Apply->Functions>

First define the displacement equations of UX, UY given and save the file.

Then Read the files UY.fun, UY.fun and names as DisplX and DisplY correspondingly.

Anslys Main Menu: Solution → Define Loads → Apply → Structural → Displacement → On nodes

Pick the left edge, select UX->instead of 'Constant value' use 'Existing Table' ->then select 'DisplX'. Repeat the same for UY also.

Step 7. Apply Loads

For Loads first find the individual nodal in MATLAB by using sigma-xy and shape functions

Ansyes Main Menu: Solution → Define Loads → Apply → Structural → Force/Moment → On nodes

->pick->top node and give $F_y = -468.8$, similarly for bottom node $F_y = -468.8$. node-22=-2500,node-21=-3437.5,Node-20=-2500

Step 8. Solve

Ansyes Main Menu: Solution → Solve → Current LS

Close Solution Done Window.

Postprocessing

Step 9. Plot Deformed Shape

ANSYS Main Menu: General Postproc → Plot Results → Deformed Shape

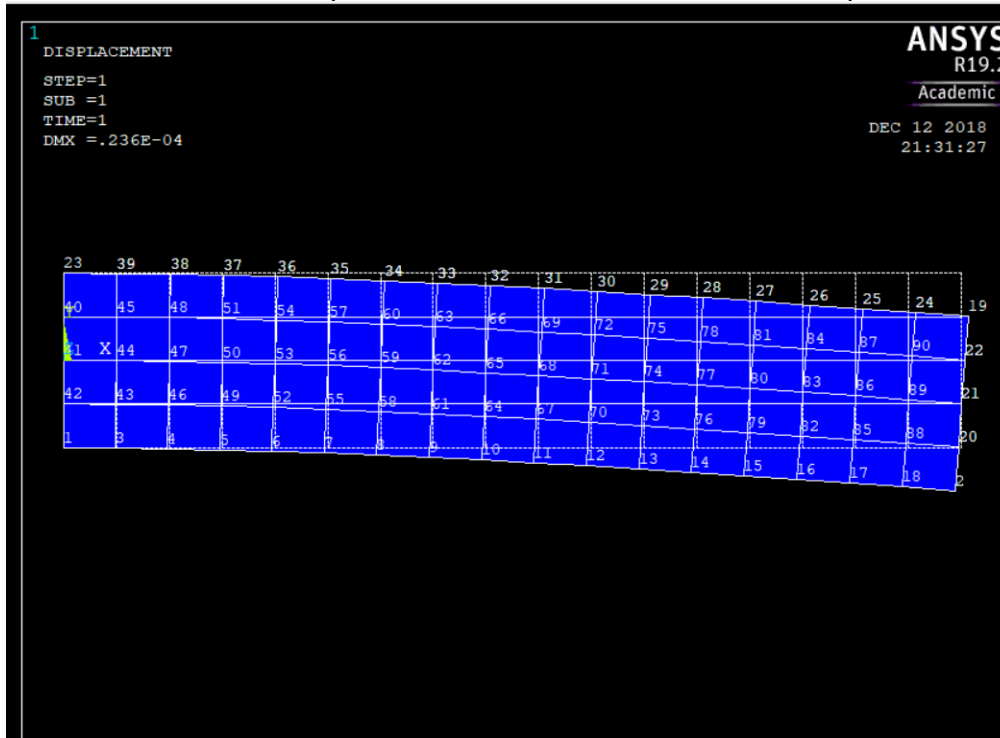
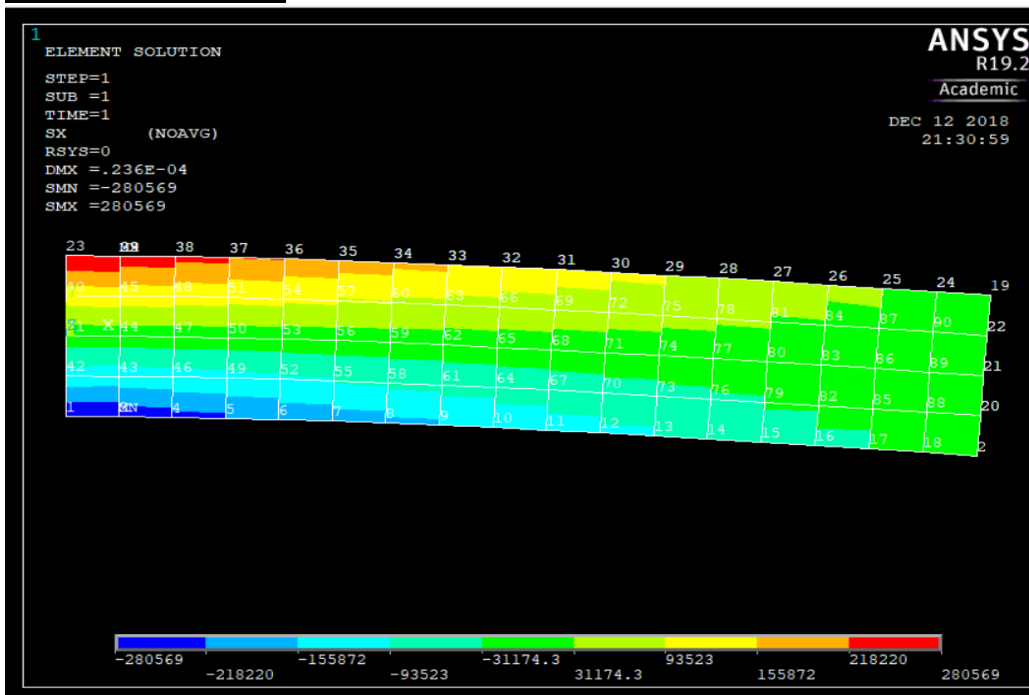


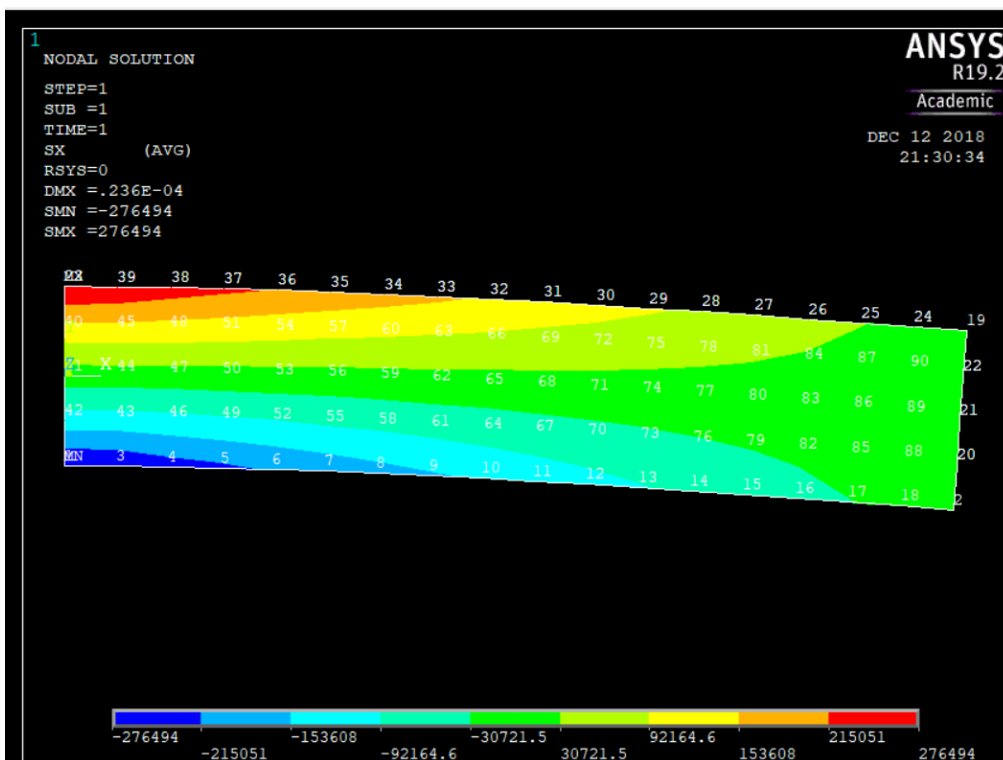
Fig.3 Deformed shape overlaid on outline of undeformed shape

Step 10. Contour Plots of Stresses ANSYS Main Menu: General Postproc → Plot Results → Contour Plot
→Element Solu Select 'Element Solution', 'Stress', and 'X-Component of stress'. OK.

Element-Stresses

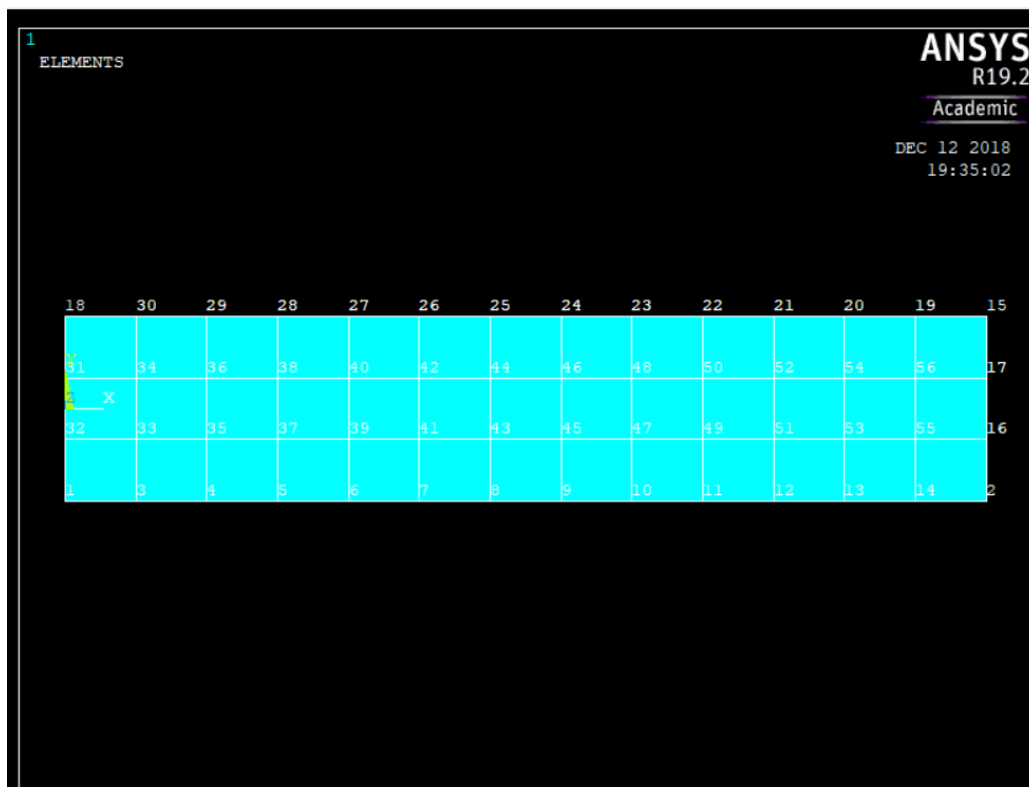


Nodal-Stresses



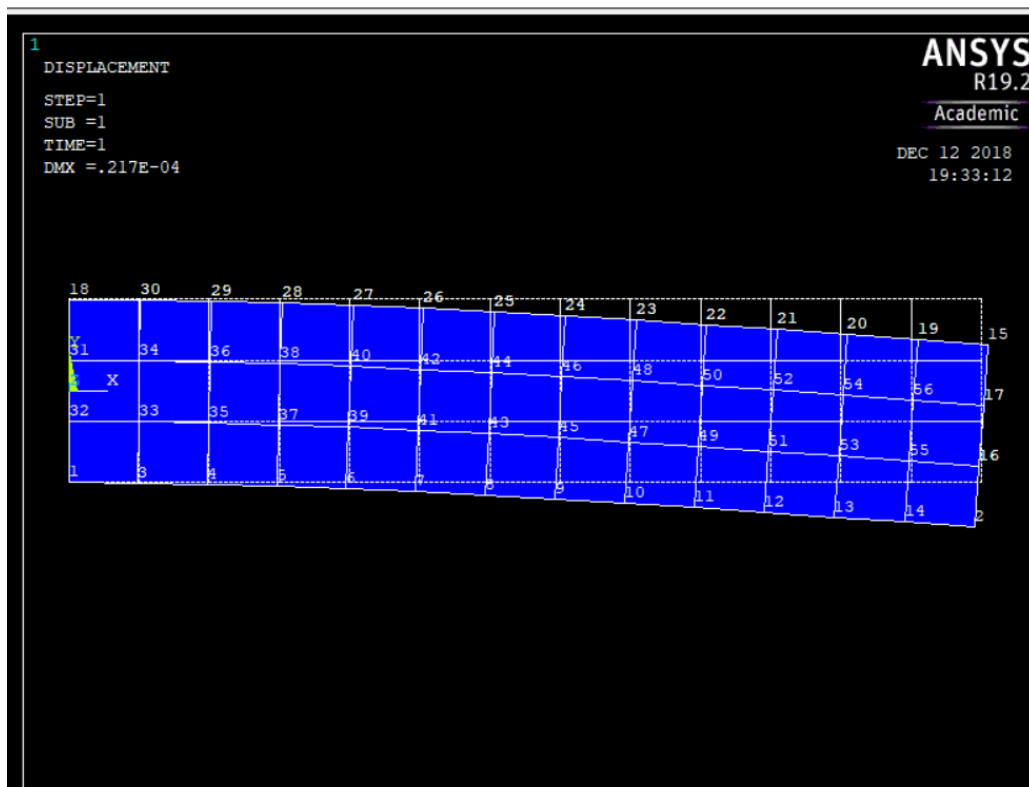
Ansys for fine Mesh:Follow the same procedure as of above

Mesh-size:0.5

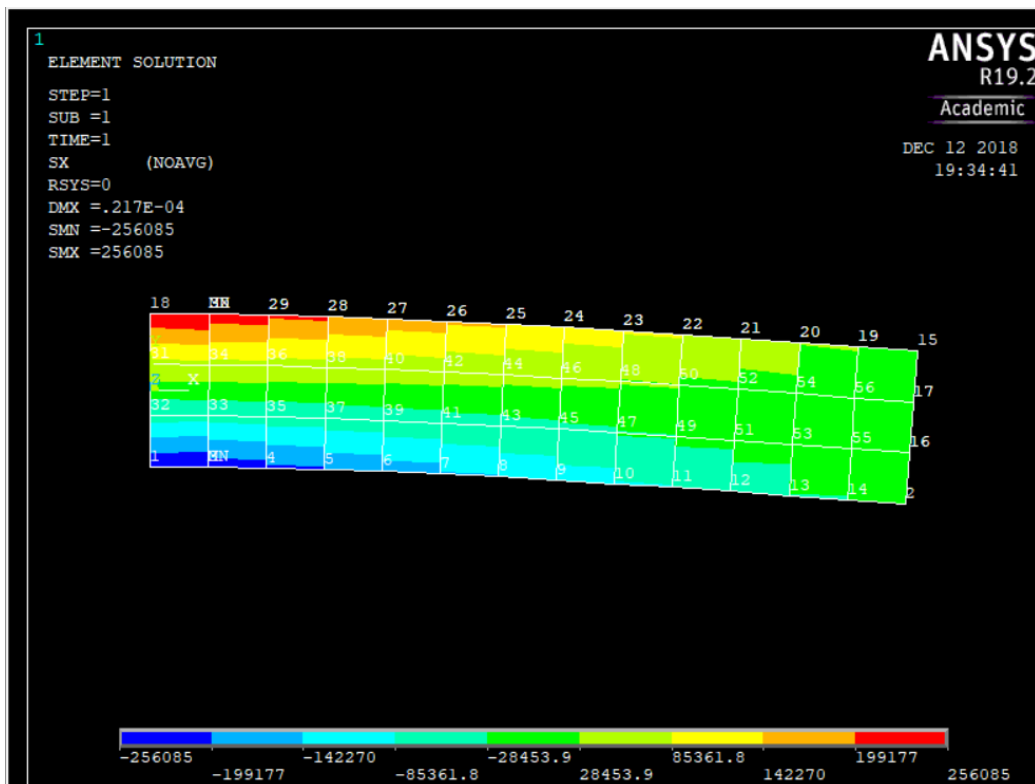


Nodal loads along y=node-15=-740.7,node 17=-3703.7,node 16=-3703.7,node-2=-740.7

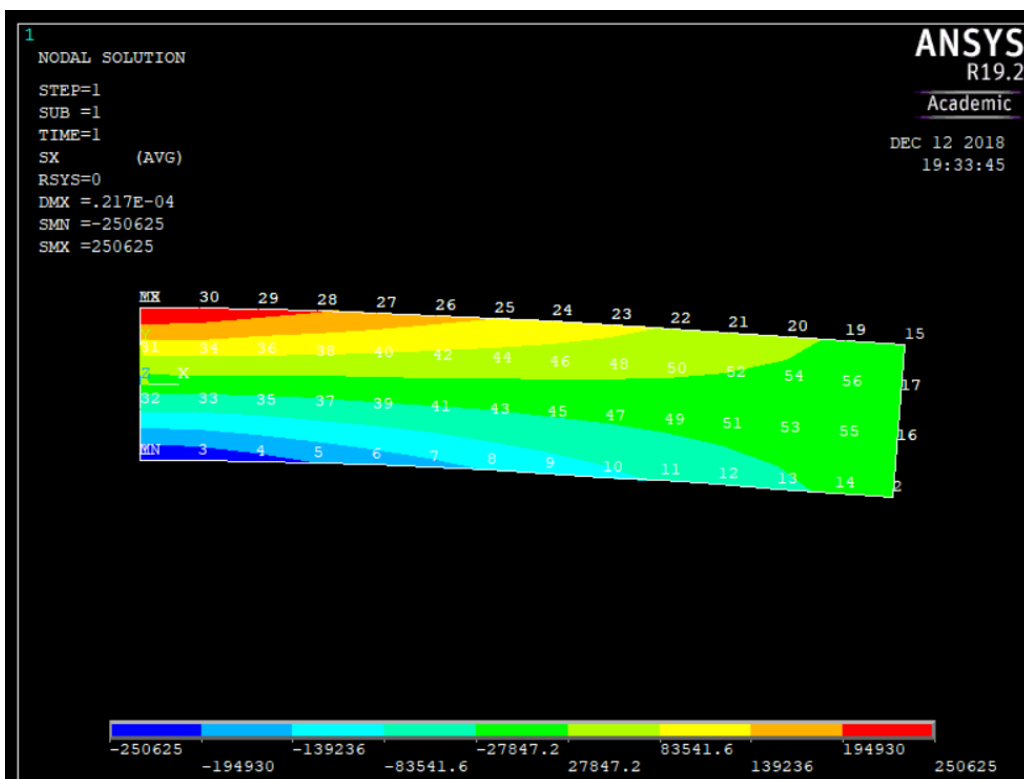
Deformed Shape



Element-Stress-X

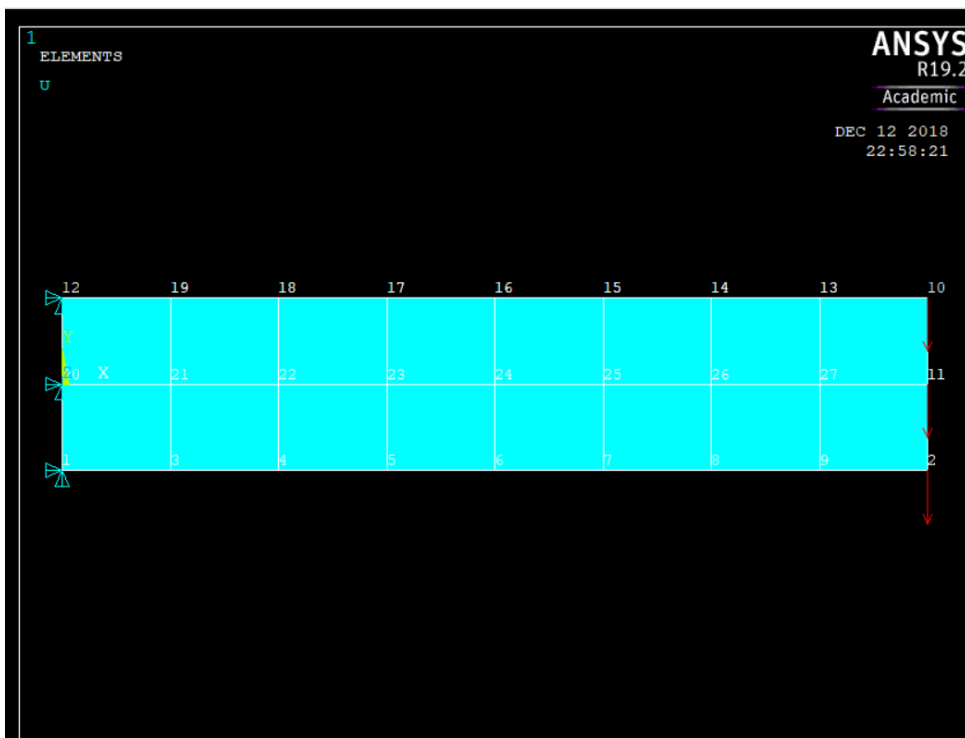


Nodal-Stress



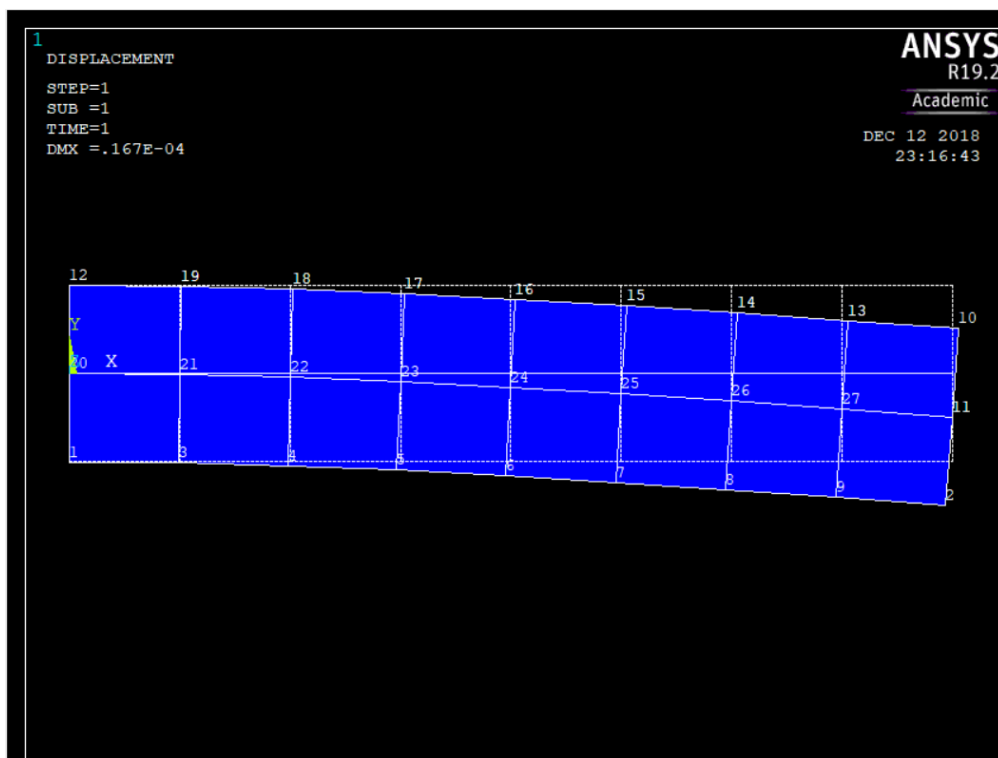
Ansys for coarse Mesh:-

Mesh-size-0.7

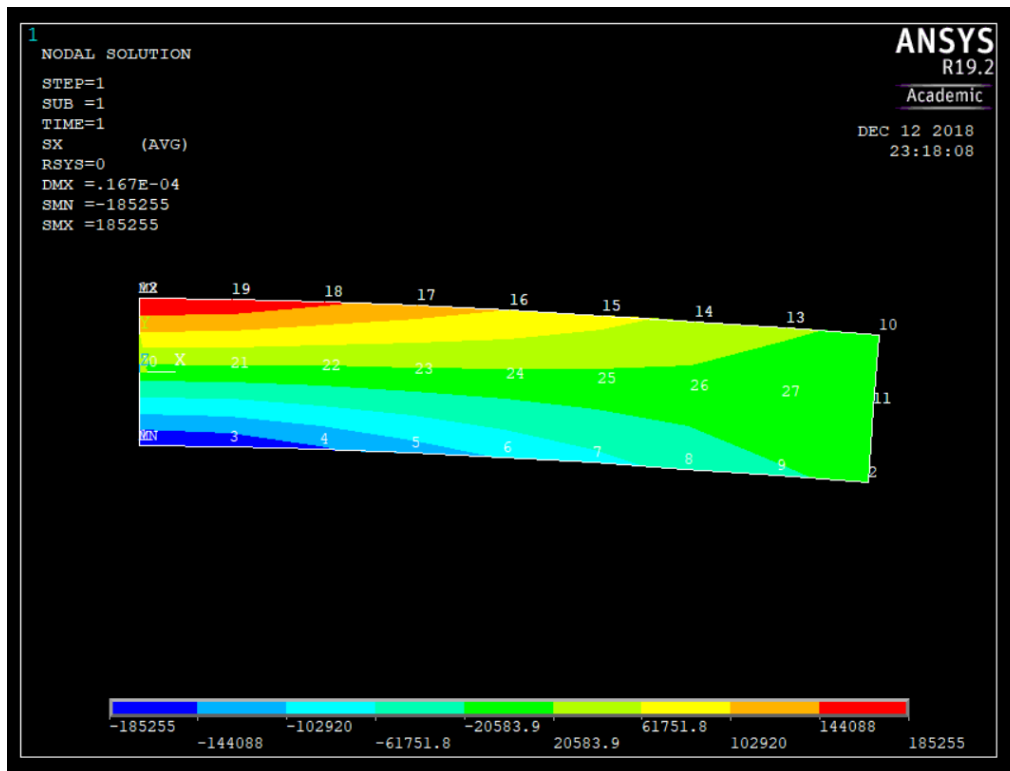


Deformed shape

Nodal-loads->node-10=-1250,node-11=-5000,node-2=-1250



Element-Stress



Observations from three types of mesh:-

- 1) As we make the mesh more fine then a very good convergence of the values can be seen.
- 2) As we decrease the mesh size the nodal loads at the end become more convergent to the applied load.

Ansys for Q8 element-Very fine mesh(0.3)

Preprocessing

Step 1. Define Element Type

ANSYS Main Menu: Preprocessor → Element Type → Add/Edit/Delete

In the pop-up window (Element Types), select Add. Then in the new window (Library of Element Types), select Structural Mass → Solid, Quad 8 node 183. OK to close.

In options, element behavior, pick 'plane stress'.

Step 2. Define Material Type

(1) ANSYS Main Menu: Preprocessor → Material Props → Material Models → Structural (double click) → Linear → Elastic → Isotropic

Input Mat'l Properties {EX=Young's '2. e11'; PRXY=0.3; OK}

Step3. Define Geometry

(1) Ansys Main Menu: Preprocessor → Modeling → Create → Areas → Rectangle → By 2 Corners
Enter X = 0; WP Y = -0.5; Width = 5; Height = 1, Click Apply to create rectangle.

Step 4. Create Mesh

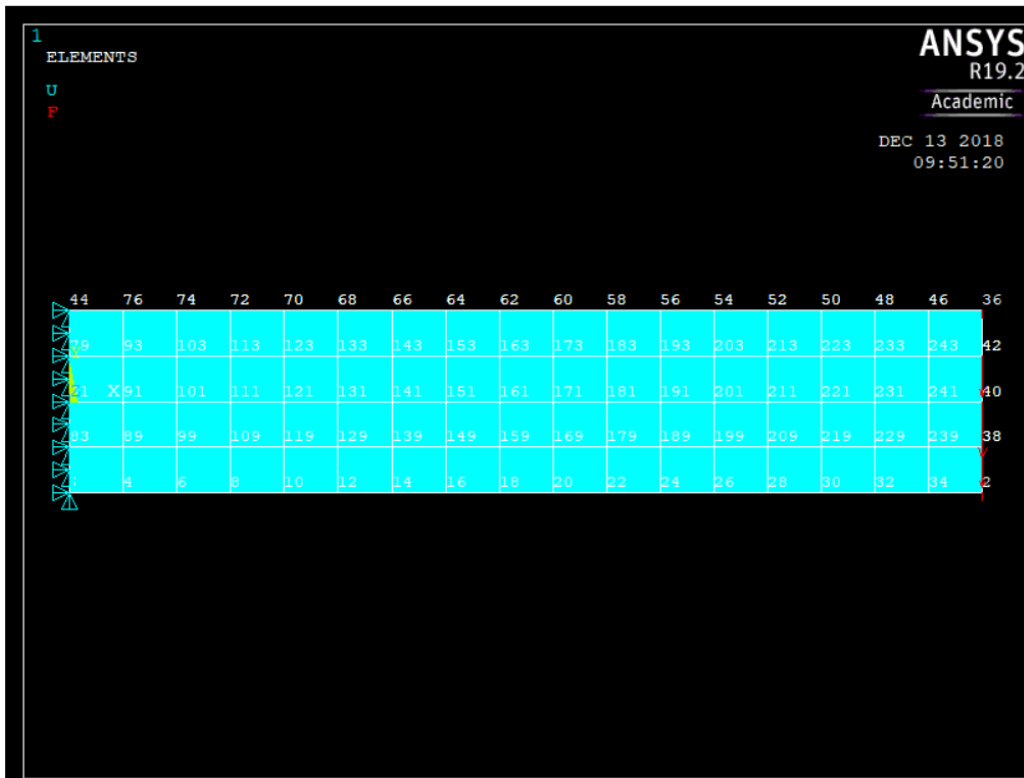
Ansyes Main Menu: Preprocessor → Meshing → Mesh Tool

A new "Mesh Tool" Window opens. Select Global.

In Size Controls panel, set global element SIZE as 0.3.

In Mesh panel, select Area, click Mesh. And pick the plate area to mesh

Mesh



Solving

Step 6. Apply Displacement Constraints:

Solve->Define Loads->Apply->Functions>

First define the displacement equations of UX, UY given and save the file.

Then Read the files UY.fun, UX.fun and names as DisplX and DisplY correspondingly.

Ansys Main Menu: Solution → Define Loads → Apply → Structural → Displacement → On nodes

Pick the left edge, select UX->instead of 'Constant value' use 'Existing Table' ->then select 'DisplX'. Repeat the same for UY also.

Step 7. Apply Loads

For Loads first find the individual nodal in MATLAB by using sigma-xy and shape functions

Ansys Main Menu: Solution → Define Loads → Apply → Structural → Force/Moment → On nodes

->pick->top node and give $F_y = -468.8$, similarly for bottom node-2 $F_y = -468.8$. node-42=-2500, node-40=-3437.5, Node-38=-2500

Step 8. Solve

Ansys Main Menu: Solution → Solve → Current LS

Close Solution Done Window.

Postprocessing

Step 9. Plot Deformed Shape

ANSYS Main Menu: General Postproc → Plot Results → Deformed Shape

Deformed Shape

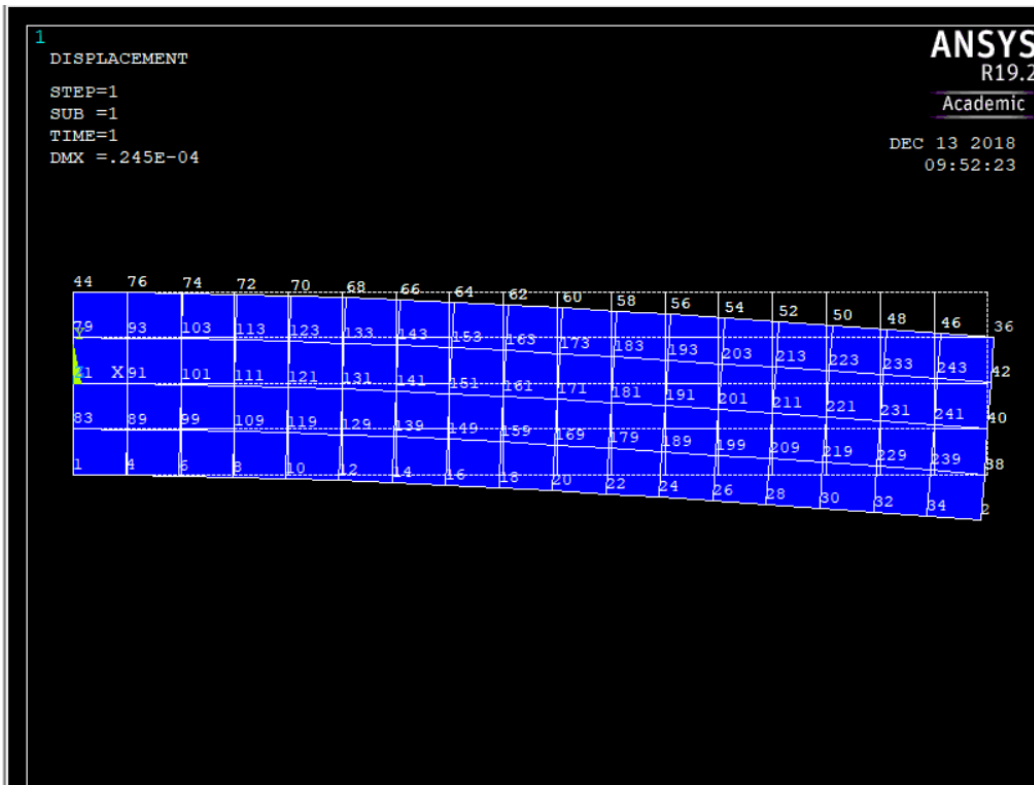
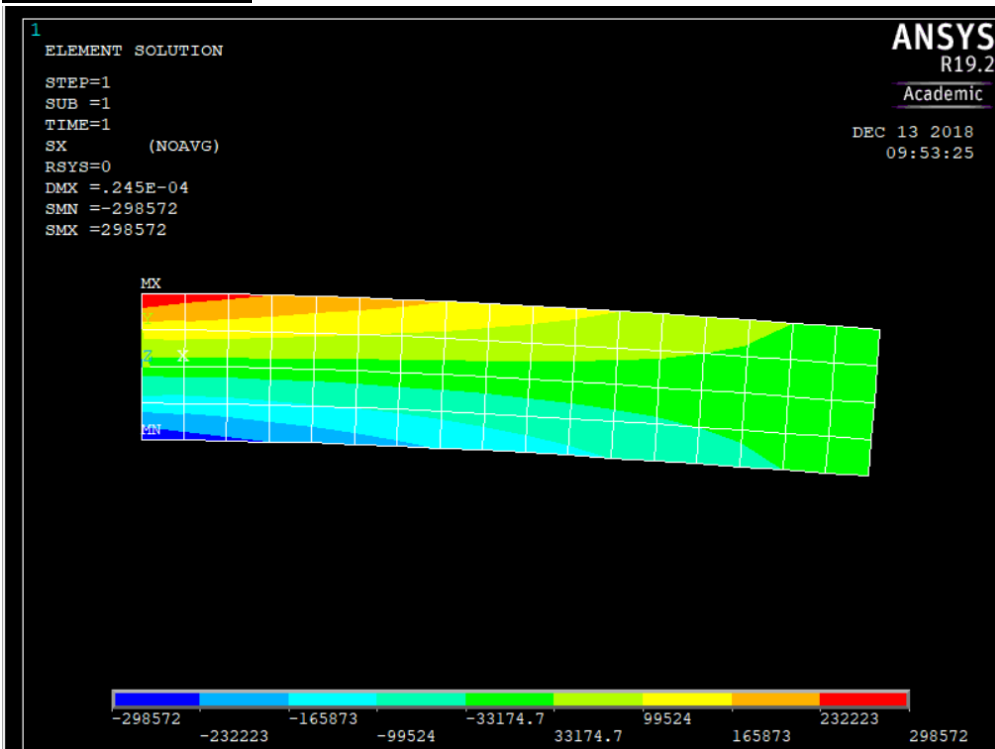


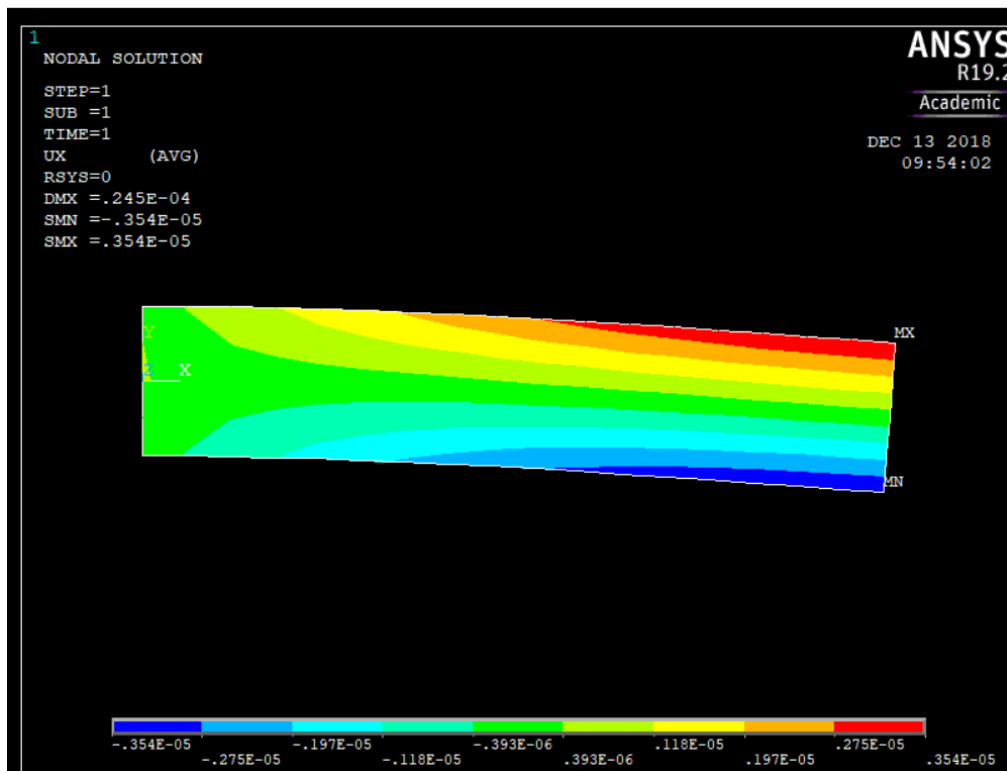
Fig.3 Deformed shape overlaid on outline of undeformed shape

Step 10. Contour Plots of Stresses ANSYS Main Menu: General Postproc → Plot Results → Contour Plot
→Element Solu Select 'Element Solution', 'Stress', and 'X-Component of stress'. OK.

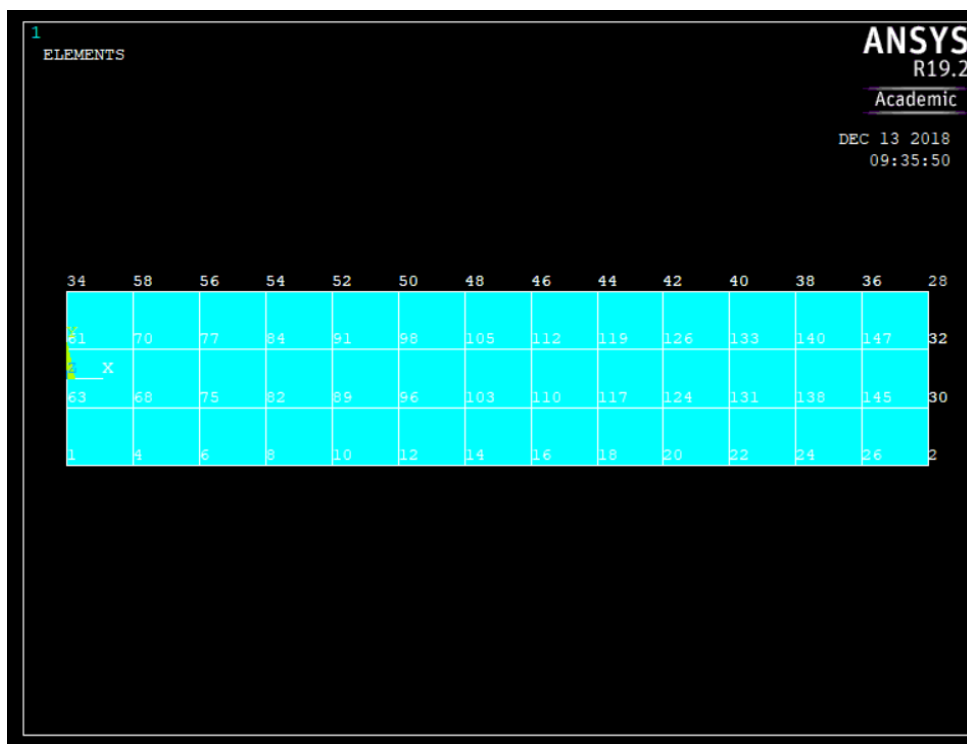
Element-Stress:-



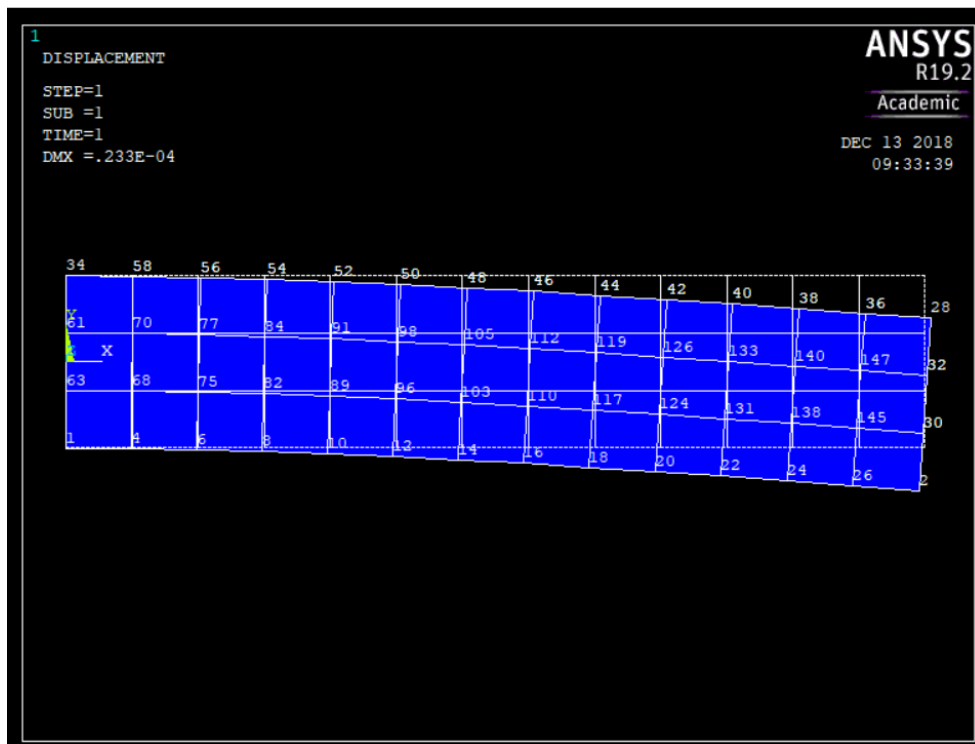
Displacement:



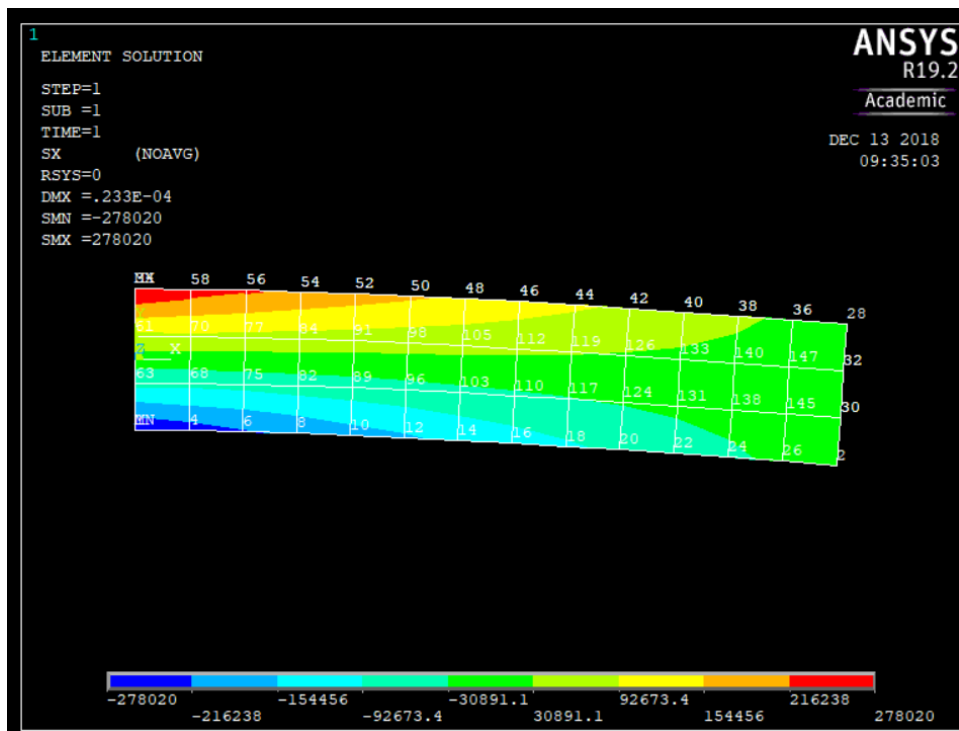
Similarly for Fine Mesh-



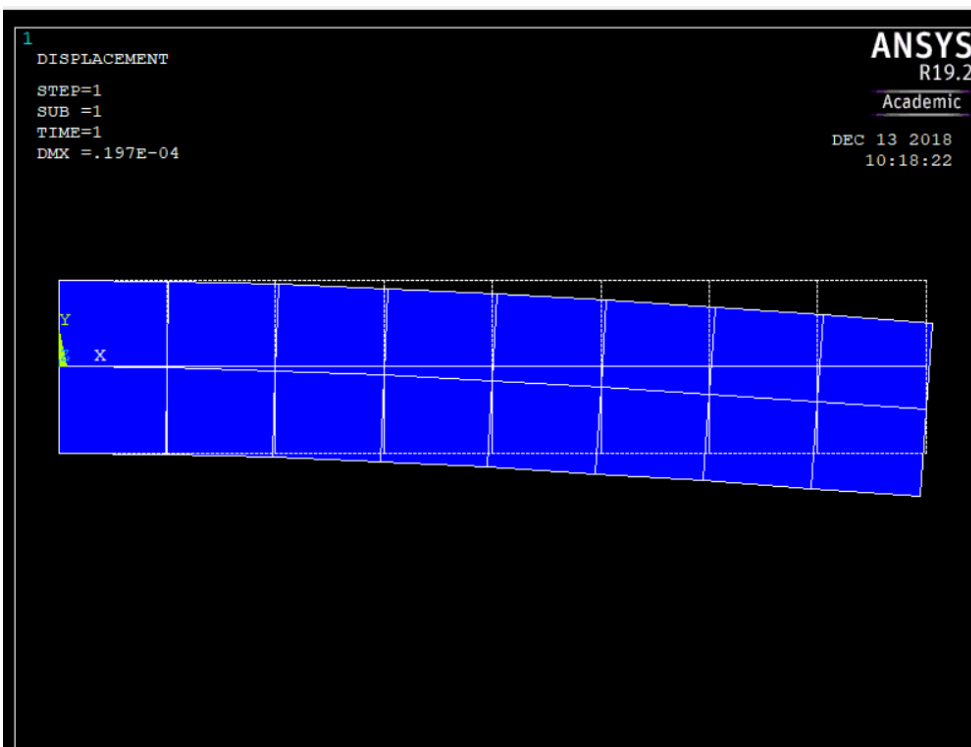
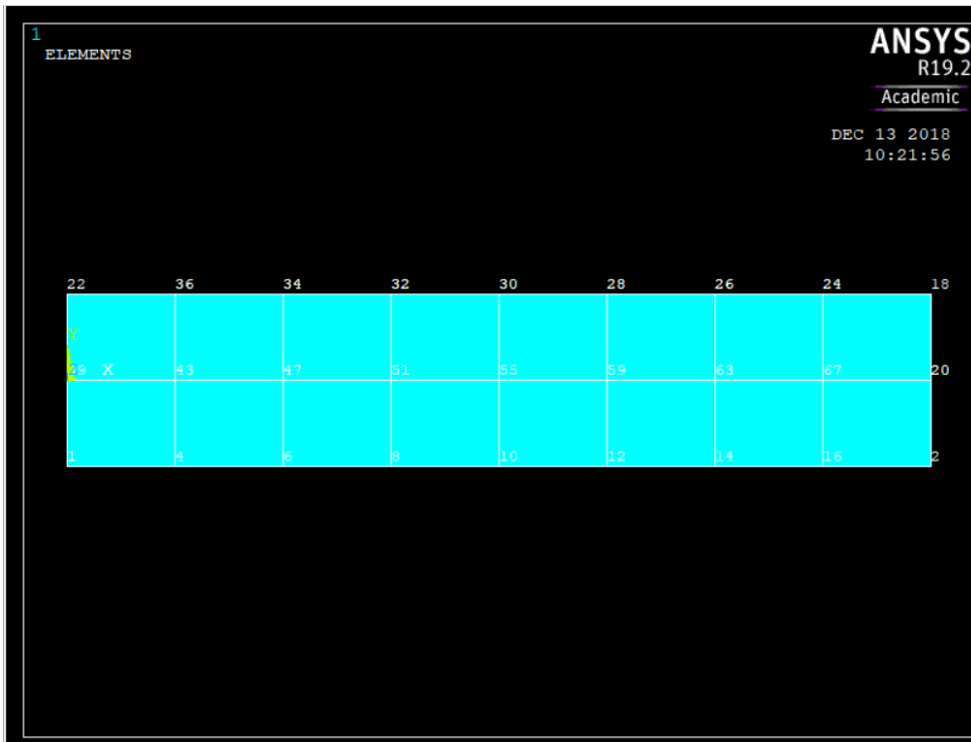
Deformed shape:-



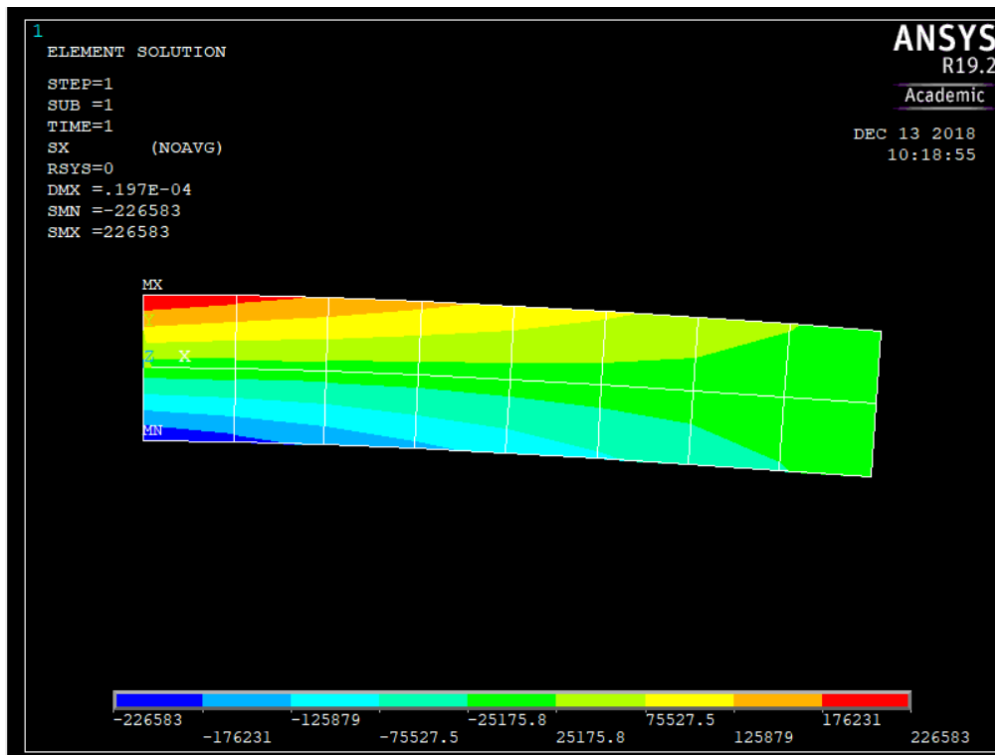
Stress-x



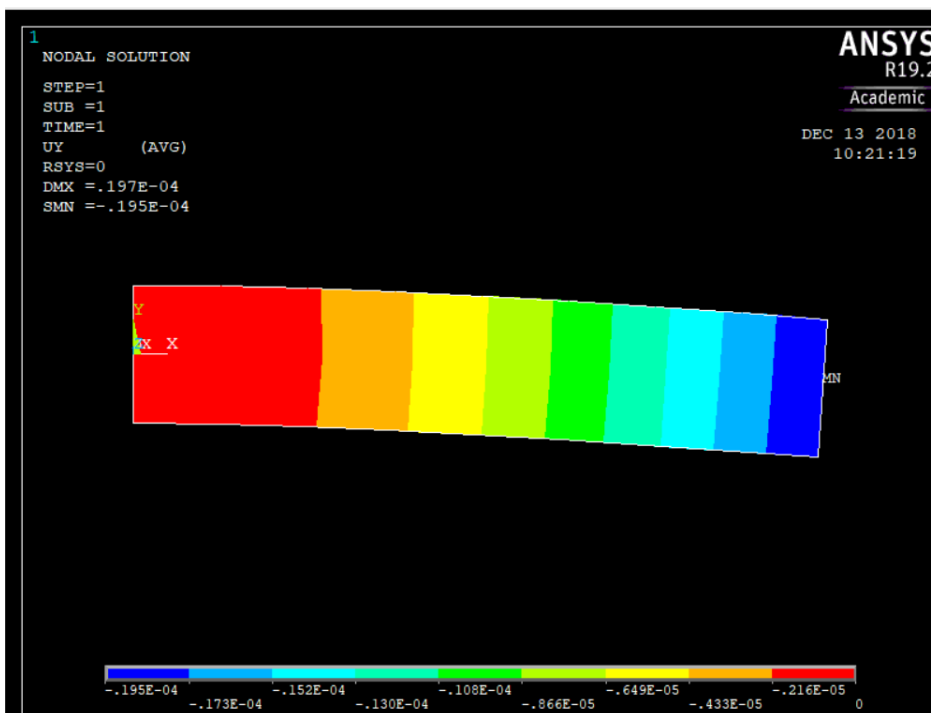
Q8 Coarse Mesh-(0.7)



Stress-X



Displacement



Sigma-x comparisons for different meshes at $x=L/2$

Considering nodes at $x=L/2$

Stress at $x=L/2$ -fine Mesh				Stress at $x=L/2$ -very fine Mesh				Stress at $x=L/2$ -very fine			
Nodes at $x=L/2$				nodes at $x=L/2$				Nodes at $x=L/2$			
46	1.23E+05			62	1.49E+05			18	-1.00E+05		
113	9.31E+04			164	1.29E+05			19	-82641		
112	41025			163	74449			20	-3.15E-05		
111	-21025			162	24563			21	82641		
110	-41025			161	1.33E-08			2	1.00E+05		
103	-9.31E+04			160	-24563						
1	-1.03E+05			159	-74449						
				158	-1.22E+05						
				20	-1.32E+05						

Observations:

- 1)As mesh size increases the values are more convergent to analytical results.
- 2)The stress values for corresponding nodes on both the sides are almost same but with different signs

2)Comparison of Ansys and Matlab Results for **Very Fine Mesh**(90 nodes) of Q4 element

Ansys-Displacement-Y

NODE	UY							
1	-1.13E-07							
2	-2.33E-05							
3	-2.53E-07							
4	-6.30E-07							
5	-1.21E-06							
6	-1.99E-06							
7	-2.94E-06							
8	-4.06E-06							
9	-5.33E-06							
10	-6.74E-06							
11	-8.27E-06							
12	-9.91E-06							
13	-1.16E-05							
14	-1.35E-05							
15	-1.54E-05							
16	-1.73E-05							
17	-1.93E-05							
18	-2.13E-05							
19	-2.33E-05							
20	-2.33E-05							
21	-2.33E-05							
22	-2.33E-05							
23	-1.13E-07							
24	-2.13E-05							
25	-1.93E-05							
26	-1.73E-05							
27	-1.54E-05							
28	-1.35E-05							
29	-1.16E-05							
30	-9.91E-06							
31	-8.27E-06							
32	-6.74E-06							
33	-5.33E-06							
34	-4.06E-06							

Matlab-Displacement Y

	-1.13E-07							
	-2.36E-05							
	-2.23E-07							
	-6.25E-07							
	-1.23E-06							
	-2.02E-06							
	-2.99E-06							
	-4.12E-06							
	-5.41E-06							
	-6.83E-06							
	-8.38E-06							
	-1.00E-05							
	-1.18E-05							
	-1.36E-05							
	-1.55E-05							
	-1.75E-05							
	-1.95E-05							
	-2.15E-05							
	-2.36E-05							
	-2.36E-05							
	-2.36E-05							
	-1.12E-07							
	-2.15E-05							
	-1.95E-05							
	-1.75E-05							
	-1.55E-05							
	-1.36E-05							
	-1.18E-05							
	-1.00E-05							
	-8.38E-06							
	-6.83E-06							
	-5.41E-06							
	-4.12E-06							

Maximum Value

34	-4.06E-06				-4.12E-06
35	-2.94E-06				-2.99E-06
36	-1.99E-06				-2.02E-06
37	-1.21E-06				-1.23E-06
38	-6.30E-07				-6.30E-07
39	-2.53E-07				-2.37E-07
40	-2.81E-08				1.25E-10
41	0				0
42	-2.81E-08				2.61E-10
43	-1.84E-07				-1.52E-07
44	-1.63E-07				-1.47E-07
45	-1.84E-07				-1.74E-07
46	-5.63E-07				-5.59E-07
47	-5.41E-07				-5.37E-07
48	-5.63E-07				-5.61E-07
49	-1.15E-06				-1.16E-06
50	-1.13E-06				-1.14E-06
51	-1.15E-06				-1.16E-06
52	-1.93E-06				-1.96E-06
53	-1.91E-06				-1.94E-06
54	-1.93E-06				-1.96E-06
55	-2.89E-06				-2.93E-06
56	-2.87E-06				-2.91E-06
57	-2.89E-06				-2.93E-06
58	-4.01E-06				-4.07E-06
59	-3.99E-06				-4.06E-06
60	-4.01E-06				-4.07E-06
61	-5.28E-06				-5.36E-06
62	-5.27E-06				-5.35E-06
63	-5.28E-06				-5.36E-06
64	-6.69E-06				-6.79E-06
65	-6.68E-06				-6.78E-06
66	-6.69E-06				-6.79E-06
67	-8.23E-06				-8.34E-06
68	-8.22E-06				-8.33E-06
69	-8.23E-06				-8.34E-06
70	-9.88E-06				-1.00E-05

33	1.63E+05	13118	-4507.2		Columns 61 through 72															
34	1.79E+05	14430	-4507.3																	
35	1.96E+05	15743	-4506.8	SX	-0.7914	-0.0393	0.7127	-0.7082	-0.0354	0.6373	-0.6244	-0.0315	0.5614	-0.5409	-0.0275	0.4858				
36	2.12E+05	17044	-4505.7	SY	-0.1181	-0.1312	-0.1443	-0.1050	-0.1181	-0.1311	-0.0918	-0.1049	-0.1181	-0.0787	-0.0918	-0.1049				
37	2.28E+05	18330	-4510.4	SXY	-0.2967	-0.3005	-0.2368	-0.2801	-0.2840	-0.2202	-0.2606	-0.2645	-0.2008	-0.2426	-0.2465	-0.1828				
38	2.45E+05	19954	-4726.1																	
39	2.67E+05	25062	-4149.7		Columns 73 through 84															
40	1.22E+05	-8512.6	-40672																	
41	-7.19E-05	4.11E-04	-39226	SX	-0.4574	-0.0236	0.4102	-0.3739	-0.0197	0.3345	-0.2903	-0.0157	0.2588	-0.2070	-0.0118	0.1833				
42	-1.22E+05	8512.6	-40672	SY	-0.0656	-0.0787	-0.0918	-0.0525	-0.0656	-0.0787	-0.0393	-0.0525	-0.0656	-0.0263	-0.0393	-0.0525				
43	-1.23E+05	-985.81	-8394.6	SXY	-0.2246	-0.2285	-0.1648	-0.2066	-0.2105	-0.1468	-0.1886	-0.1926	-0.1288	-0.1725	-0.1762	-0.1126				
44	-0.36633E-004-	4.07E-05	-11150																	
45	1.23E+05	985.81	-8394.6		Columns 85 through 90															
46	-1.18E+05	144.38	-8603.1																	
47	-1.12E-05	4.42E-05	-11665	SX	-0.1249	-0.0080	0.1092	-0.0392	-0.0042	0.0309										
48	1.18E+05	-144.38	-8603.1	SY	-0.0132	-0.0267	-0.0388	0.0009	-0.0139	-0.0284										
49	-1.11E+05	18.504	-8677.4	SXY	-0.1523	-0.1561	-0.0925	-0.1340	-0.1404	-0.0761										
50	-3.57E-05	2.07E-04	-12050																	
51	1.11E+05	-18.504	-8677.4																	
52	-1.03E+05	-1.5695	-8670.2																	
53	-5.41E-05	6.39E-05	-12059																	
54	1.03E+05	1.5691	-8670.2																	
55	-95498. -	0.35601	-8670.1																	
56	-2.39E-05	8.32E-05	-12052																	
57	95498	0.35634	-8670.1																	
58	-87540	1.98E-02	-8669.9																	
59	-0.50137E-004-	4.34E-06	-12052																	
60	-87540	1.97E-02	-8669.9																	
61	-79582	5.15E-03	-8670																	
62	2.66E-05	8.86E-05	-12052																	
63	-79582	5.20E-03	-8670																	
64	-71624	9.47E-05	-8670																	
65	-8.92E-06	1.09E-05	-12052																	
66	-71624	1.65E-04	-8670																	
67	-63665. -	3.60E-04	-8670																	

67	-63665. -	3.60E-04	-8670
68	-0.13377E-004-	8.53E-05	-12052
69	63665	2.71E-04	-8670
70	-55707	2.17E-06	-8670
71	-0.13418E-004-	8.54E-05	-12052
72	55707	2.16E-04	-8670
73	-47749. -	7.37E-04	-8670
74	-0.28827E-004-	5.54E-05	-12052
75	47749	8.34E-04	-8670
76	-39791	3.78E-02	-8670
77	-2.49E-06	3.24E-05	-12052
78	-39791	3.78E-02	-8670
79	-31831. -	1.54E-02	-8669.8
80	1.50E-05	5.00E-05	-12053
81	31831	1.55E-02	-8669.8
82	-23877	-1.8416	-8671.2
83	-8.69E-06	1.17E-05	-12049
84	23877	1.8415	-8671.2
85	-15997	-5.4264	-8659
86	1.27E-06	1.44E-05	-12027
87	15997	5.4264	-8659
88	-7980.9	125.08	-8687.4
89	-1.36E-06	5.38E-07	-12092
90	7980.9	-125.08	-8687.4

MINIMUM VALUES

NODE 1 3 40
VALUE -0.27649E+006 - 25062. -40672

MAXIMUM VALUES

NODE 23 39 3
VALUE 2.76E+05 25062. -4149.7

Comparison of Ansys and Matlab Results for **Fine-Mesh** of Q4 element

Ansys-Y-Displacement			Matlab Y-Displacement		
NODE	UY				
1	-1.13E-07		-1.13E-07		
2	-2.15E-05	Max	-2.13E-05	Max	
3	-3.21E-07		-3.49E-07		
4	-8.92E-07		-8.80E-07		
5	-1.77E-06		-1.75E-06		
6	-2.94E-06		-2.90E-06		
7	-4.36E-06		-4.30E-06		
8	-6.00E-06		-5.94E-06		
9	-7.85E-06		-7.77E-06		
10	-9.87E-06		-9.77E-06		
11	-1.20E-05		-1.19E-05		
12	-1.43E-05		-1.42E-05		
13	-1.67E-05		-1.65E-05		
14	-1.91E-05		-1.89E-05		
15	-2.15E-05		-2.13E-05		
16	-2.15E-05		-2.13E-05		
17	-2.15E-05		-2.13E-05		
18	-1.13E-07		-1.13E-07		
19	-1.91E-05		-1.89E-05		
20	-1.67E-05		-1.65E-05		
21	-1.43E-05		-1.42E-05		
22	-1.20E-05		-1.19E-05		
23	-9.87E-06		-9.77E-06		
24	-7.85E-06		-7.77E-06		
25	-6.00E-06		-5.94E-06		
26	-4.36E-06		-4.30E-06		
27	-2.94E-06		-2.90E-06		
28	-1.77E-06		-1.75E-06		
29	-8.92E-07		-8.87E-07		
30	-3.21E-07		-3.27E-07		
31	-1.25E-08		-8.61E-09		
32	-1.25E-08		-1.13E-07		
33	-2.48E-07		-2.38E-07		
34	-2.48E-07		-2.61E-07		
35	-8.21E-07		-8.15E-07		
36	-8.21E-07		-8.12E-07		
37	-1.71E-06		-1.68E-06		
38	-1.71E-06		-1.69E-06		
39	-2.88E-06		-2.84E-06		
40	-2.88E-06		-2.84E-06		
41	-4.31E-06		-4.25E-06		
42	-4.31E-06		-4.25E-06		
43	-5.96E-06		-5.89E-06		
44	-5.96E-06		-5.89E-06		
45	-7.81E-06		-7.73E-06		
46	-7.81E-06		-7.73E-06		
47	-9.84E-06		-9.74E-06		
48	-9.84E-06		-9.74E-06		
49	-1.20E-05		-1.19E-05		
50	-1.20E-05		-1.19E-05		
51	-1.43E-05		-1.42E-05		
52	-1.43E-05		-1.42E-05		
53	-1.66E-05		-1.65E-05		
54	-1.66E-05		-1.65E-05		
55	-1.91E-05		-1.89E-05		
56	-1.91E-05		-1.89E-05		
MAXIMUM	ABSOLUTE VALUES				
NODE	16				
VALUE	-2.15E-05				

Stress-X values Comparision at nodes:-

Ansys Stress Results					MATLAB RESULTS FOR STRESSES AT NODES													
	NODE	SX	SY	SKY		NODES 1	TO	NODES 12										
Maximum Stress	1	-2.51E+05	-15163	-43576		1.0e+05*												
	2	-10773	-5139.6	-4647	SX	-2.7532	-0.1077	-2.2369	-2.0848	-1.8774	-1.6802	-1.4818	-1.2834	-1.0850	-0.8866	-0.6883	-0.4898	
	3	-2.42E+05	-29253	-3879.5	SY	-1.5016	-0.0514	-0.0085	-0.2381	-0.1734	-0.1594	-0.1369	-0.1161	-0.0950	-0.0739	-0.0528	-0.0317	
	4	-2.18E+05	-22967	-5188.1	SKY	-0.6979	-0.0465	-0.3489	-0.3707	-0.3331	-0.3055	-0.2770	-0.2487	-0.2203	-0.1919	-0.1646	-0.1350	
	5	-1.98E+05	-21122	-4958.8														
	6	-1.79E+05	-18997	-4979.5														
	7	-1.59E+05	-16887	-4978														
	8	-1.39E+05	-14776	-4978.1	SX	-0.2910	-0.0877	0.1077	-0.0350	0.0413	2.4782	0.0877	0.2910	0.4898	0.6883	0.8866	1.0850	
	9	-1.19E+05	-12665	-4978.1	SY	-0.0107	0.0153	0.0514	-0.0105	0.0315	0.1191	-0.0153	0.0107	0.0317	0.0528	0.0739	0.0950	
	10	-99211	-10554	-4978.1	SKY	-0.1072	-0.0733	-0.0465	-0.1124	-0.0667	-0.4492	-0.0733	-0.1072	-0.1350	-0.1646	-0.1919	-0.2203	
	11	-79369	-8443.5	-4978.2														
	12	-59529	-6331.4	-4976														
	13	-39676	-4247.2	-5002.8														
	14	-19512	-1695.3	-4803	SX	1.2834	1.4818	1.6803	1.8803	2.0756	2.2462	0.5693	-0.5560	-0.8767	0.7251	-0.7287	0.5883	
	15	10773	5139.6	-4647	SY	0.1160	0.1375	0.1573	0.1849	0.1762	0.2761	-0.4535	0.4571	-0.3989	-0.1802	-0.1994	-0.2699	
	16	-3818.2	-2099.3	-8961.6	SKY	-0.2487	-0.2771	-0.3055	-0.3356	-0.3597	-0.3682	-0.5247	-0.2311	-0.5135	-0.3519	-0.4165	-0.3822	
	17	3818.2	2099.3	-8961.6														
	18	2.51E+05	15163	-43576														
	19	19512	1695.3	-4803														
	20	39676	4247.2	-5002.8	SX	-0.6708	0.5436	-0.5978	0.4836	-0.5278	0.4265	-0.4573	0.3687	-0.3870	0.3110	-0.3166	0.2533	
	21	59529	6331.4	-4976	SY	-0.2063	-0.2161	-0.1781	-0.2017	-0.1585	-0.1791	-0.1372	-0.1584	-0.1161	-0.1372	-0.0950	-0.1161	
	22	79369	8443.5	-4978.2	SKY	-0.3977	-0.3464	-0.3671	-0.3187	-0.3389	-0.2898	-0.3105	-0.2615	-0.2821	-0.2331	-0.2536	-0.2047	
	23	99211	10554	-4978.1														
	24	1.19E+05	12665	-4978.1														
	25	1.39E+05	14776	-4978.1														
	26	1.59E+05	16887	-4978	SX	-0.2463	0.1957	-0.1757	0.1377	-0.1070	0.0818	-0.0350	0.0213					
	27	1.79E+05	18997	-4979.5	SY	-0.0739	-0.0950	-0.0527	-0.0739	-0.0321	-0.0520	-0.0105	-0.0352					
	28	1.98E+05	21122	-4958.8	SKY	-0.2264	-0.1774	-0.1968	-0.1479	-0.1681	-0.1189	-0.1407	-0.0935					
	29	2.18E+05	22967	-5188.1														
	30	2.42E+05	29253	-3879.5														
	31	72356	-8305.1	-46148														
	32	-72356	8305.1	-46148														
	33	-73870	-430.75	-8626.3														
	34	73870	430.75	-8626.3														
	35	-69914	284.33	-8577.3														
	36	69914	-284.33	-8577.3														
	37	-64100	-30.233	-8728.8														
	38	64100	30.232	-8728.8														

39	-57621	2.2853	-8712
40	57621	-2.2854	-8712
41	-51224	0.13827	-8713.3
42	51224	0.13887	-8713.3
43	-44821	0.69305E	-8713.2
44	-44821	0.66073E	-8713.2
45	-38418	-0.10975E	-8713.2
46	38418	0.75021E	-8713.2
47	-32015	0.13904E	-8713.2
48	-32015	0.13586E	-8713.2
49	-25613	0.23881	-8713.1
50	25613	0.23842	-8713.1
51	-19201	3.3454	-8714.9
52	19201	-3.3454	-8714.9
53	-12876	-35.145	-8695.9
54	12876	35.145	-8695.9
55	-6417.6	153.83	-8763.6
56	6417.6	-153.83	-8763.6
MINIMUM VALUES			
NODE	1	3	31
VALUE	-0.25062E+	29253	-46148
MAXIMUM VALUES			
NODE	18	30	3
VALUE	2.51E+05	29253	-3879.5

Comparisons between Ansys and Matlab for Coarse Mesh(27 Nodes)

A	B	C	D	E	F	G	H
Ansys Displacement-Y					MATLAB DISPLACEMENT-Y		
NODE	UY						
1	-1.13E-07				-1.13E-07		
2	-1.65E-05				-1.69E-05		
3	-5.00E-07				-5.52E-07		
4	-1.58E-06				-1.67E-06		
5	-3.21E-06				-3.36E-06		
6	-5.30E-06				-5.52E-06		
7	-7.78E-06				-8.05E-06		
8	-1.05E-05				-1.09E-05		
9	-1.35E-05				-1.39E-05		
10	-1.65E-05				-1.69E-05		
11	-1.65E-05				-1.68E-05		
12	-1.13E-07				-1.13E-07		
13	-1.35E-05				-1.39E-05		
14	-1.05E-05				-1.09E-05		
15	-7.78E-06				-8.05E-06		
16	-5.30E-06				-5.52E-06		
17	-3.21E-06				-3.36E-06		
18	-1.58E-06				-1.67E-06		
19	-5.00E-07				-5.31E-07		
20	0				2.15E-18		
21	-4.46E-07				-4.75E-07		
22	-1.52E-06				-1.61E-06		
23	-3.16E-06				-3.31E-06		
24	-5.27E-06				-5.48E-06		
25	-7.75E-06				-8.02E-06		
26	-1.05E-05				-1.08E-05		
27	-1.35E-05				-1.38E-05		
MAXIMUM ABSOLUTE VALUES							
NODE	11						
VALUE -	1.65E-05						
VALUE -	0.16505E-004						

[illegible]

3) Sigma-X at $x=L/2$ for three types of mesh:

The nodes which are at $x=L/2$ are Considered

Sigma-X at $x=L/2$:Very Fine Mesh

Ansys_STRESS-X at $x=L/2$		MATLAB STRESS-X at $X=L/2$	
Nodes at $X=L/2$			
32	1.47E+05	1.41E+05	
66	71624	70624	
65	8.92E-06	9.82E-06	
64	1.19E+05	1.19E+05	
10	-1.47E+05	-1.40E+05	

Sigma-X at $x=L/2$:Fine Mesh

ANSYS Sigma X at $x=L/2$		MATLAB STRESS-X at $X=L/2$ 2-Point	
Nodes at $x=L/2$			
9	-1.19E+05	-1.25E+05	
45	-38418	-45418	
46	38418	45418	
24	1.19E+05	1.25E+05	

Sigma-X at $x=L/2$:Coarse Mesh

Ansys Stress at $x=L/2$		Matlab Stress at $x=L/2$ ---2point Gauss	
Nodes at $x=L/2$			
6	-1.00E+05	-9.00E+04	
24	-3.15E-05	-2.90E+05	
16	1.00E+05	9.00E+04	

3)Comparison of Stress(sigma-x) at 2-point,1-point,3-point Gauss Quadrature

ANSYS Sigma X at $x=L/2$		MATLAB STRESS-X at $X=L/2$ 2-Point	MATLAB STRESS-X at $X=L/2$ 1-Point	MATLAB STRESS-X at $X=L/2$ 3-Point
Nodes at $x=L/2$				
9	-1.19E+05	-1.25E+05	3.10E+05	-1.09E+05
45	-38418	-45418	-68418	-38700
46	38418	45418	68418	38700
24	1.19E+05	1.25E+05	-309930.00	1.09E+05