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# FORMATION TRACKING CONTROL FOR MULTI-ROBOT SYSTEMS BASED ON REINFORCEMENT LEARNING - ADP APPROACH

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#### **Table of contents**

- 1. Introduction
- 2. System modeling
- 3. Multi-agent control design using traditional method
- 4. Multi-agent control design using adaptive dynamic programming approach
- 5. Simulation results
- 6. Conclusion



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### Introduction

#### 1. Introduction

- In this research, a system of multiple wheeled mobile robots is required to track and maintain specific formations while following desired trajectories.
- The objectives of this work are:
  - Introduce the multi-agent consensus control algorithms for single- and doubleintegrator models, and ensure formation tracking of wheeled mobile robot system in case of no disturbance by traditional control method.
  - Present an ADP-based adaptive optimal controller for the wheeled mobile robot formation tracking, considering wheel slip and external disturbances.



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### System modeling

#### A. Introduction of wheeled mobile robot (WMR)

- Two main driving wheels, one free wheel

Symbol	Meaning
$G(x_G, y_G)$	the center of mass of the platform
$M(x_M, y_M)$	the midpoint of the wheel shaft
$F_1, F_2$	total longitudinal friction forces at the right and left wheel
$F_3$	total lateral friction force along the wheel shaft
$F_4$ , $\mu$	external force and moment acting on G, respectively
$\gamma_R, \gamma_L$	longitudinal slip factors of the left and right wheels
η	the lateral slip factor along the wheel shaft
а	the distance between point M and G
b	half of the wheel shaft
heta	the orientation of the WMR
r	the radius of each wheel

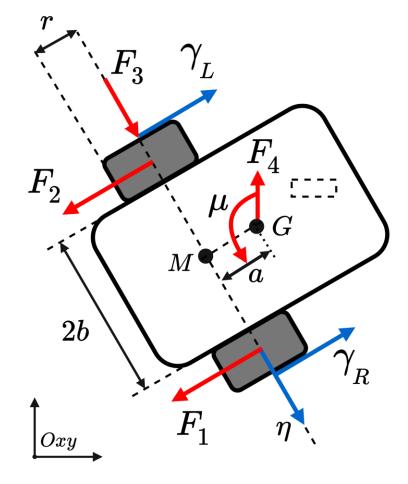


Diagram of the WMR



#### B. WMR system model:

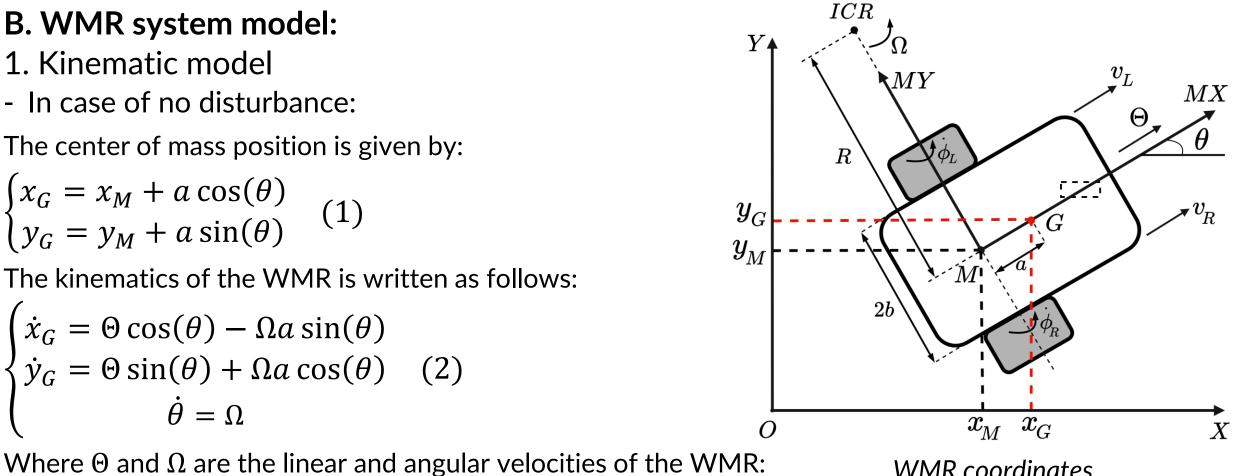
- 1. Kinematic model
- In case of no disturbance:

The center of mass position is given by:

$$\begin{cases} x_G = x_M + a\cos(\theta) \\ y_G = y_M + a\sin(\theta) \end{cases}$$
 (1)

The kinematics of the WMR is written as follows:

$$\begin{cases} \dot{x}_G = \Theta \cos(\theta) - \Omega a \sin(\theta) \\ \dot{y}_G = \Theta \sin(\theta) + \Omega a \cos(\theta) \\ \dot{\theta} = \Omega \end{cases}$$
 (2)



WMR coordinates

 $\Theta = \frac{r(\dot{\phi}_R + \dot{\phi}_L)}{2}$ ,  $\Omega = \frac{r(\dot{\phi}_R - \dot{\phi}_L)}{2h}$ ,  $\dot{\phi}_R$  and  $\dot{\phi}_L$  are the angular velocities of the right and left wheel.



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#### B. WMR system model:

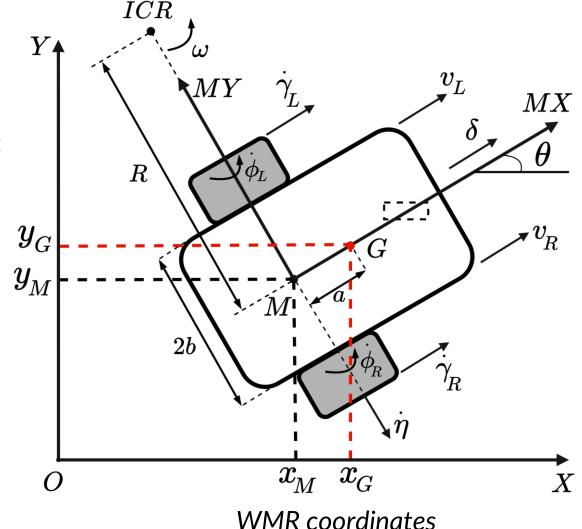
- 1. Kinematic model
- In case of wheel slip and external disturbances

The actual linear and angular velocities are computed as:

$$\begin{cases} \delta = \frac{r(\dot{\phi}_R + \dot{\phi}_L)}{2} + \frac{\dot{\gamma}_R + \dot{\gamma}_L}{2} \\ \omega = \frac{r(\dot{\phi}_R - \dot{\phi}_L)}{2b} + \frac{\dot{\gamma}_R - \dot{\gamma}_L}{2b} \end{cases}$$
(3)

The actual kinematics of the WMR is written as follows:

$$\begin{aligned}
\dot{x}_G &= \delta \cos(\theta) - \omega a \sin(\theta) - \dot{\eta} \sin(\theta) \\
\dot{y}_G &= \delta \sin(\theta) + \omega a \cos(\theta) + \dot{\eta} \cos(\theta) \\
\dot{\theta} &= \omega
\end{aligned} \tag{4}$$





#### B. WMR system model:

- 1. Kinematic model
- In case of wheel slip and external disturbances

Combining (3) and (4), the position of the WMR can be

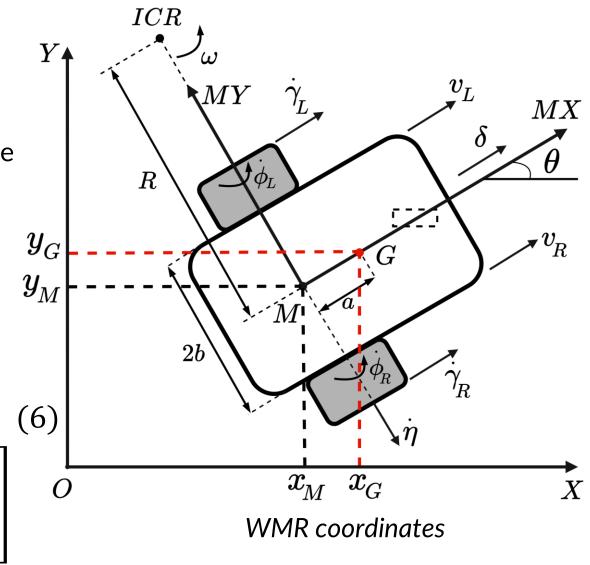
rewritten as:

$$\dot{p} = Sv + d_p \tag{5}$$

Where,

$$S = \begin{bmatrix} x_G & y_G \end{bmatrix}^T, & v = [\dot{\phi}_R & \dot{\phi}_L]^T \\ S = \begin{bmatrix} \frac{r\cos(\theta)}{2} - \frac{ar\sin(\theta)}{2b} & \frac{r\cos(\theta)}{2} + \frac{ar\sin(\theta)}{2b} \\ \frac{r\sin(\theta)}{2} + \frac{ar\cos(\theta)}{2b} & \frac{r\sin(\theta)}{2} - \frac{ar\cos(\theta)}{2b} \end{bmatrix}$$

$$d_p = \begin{bmatrix} \frac{\cos(\theta)(\dot{\gamma}_R + \dot{\gamma}_L)}{2} - \frac{a\sin(\theta)(\dot{\gamma}_R - \dot{\gamma}_L)}{2b} - \dot{\eta}\sin(\theta) \\ \frac{2}{2} \frac{\sin(\theta)(\dot{\gamma}_R + \dot{\gamma}_L)}{2} + \frac{a\cos(\theta)(\dot{\gamma}_R - \dot{\gamma}_L)}{2b} + \dot{\eta}\cos(\theta) \end{bmatrix}$$



#### B. WMR system model:

- 2. Dynamic model
- In case of no disturbance

The dynamic model is derived by Lagrange formulation:

$$\frac{d}{dt} \left( \frac{\partial L}{\partial \dot{q}} \right) - \frac{\partial L}{\partial q} = F - A^T \lambda \quad (7)$$

Where,

+L defines the Lagrangian (difference between kinetic and potential energy of the system), given as:

$$L = K - U \quad (8)$$

- U = 0: potential energy of the system
- K: kinetic energy of the WMR platform, derived as:  $K = K_G = \frac{1}{2} m_G (\dot{x}_G^2 + \dot{y}_G^2) + \frac{1}{2} I_G \Omega^2$  (9)

 $m_G$ : mass of the platform of the WMR without the driving wheels

 $I_G$ : moment of inertia of the WMR platform about the vertical axis through G



#### B. WMR system model:

- 2. Dynamic model
- In case of no disturbance

The dynamic model is derived by Lagrange formulation:

$$\frac{d}{dt} \left( \frac{\partial L}{\partial \dot{q}} \right) - \frac{\partial L}{\partial q} = F - A^T \lambda \quad (7)$$

Where,

 $+ q = [x_G \quad y_G \quad \theta]^T$  is the general coordinate,  $\lambda = [\lambda_1 \quad \lambda_2 \quad \lambda_3]^T$  is the Lagrange multiplier.

 $A(q) = [-\sin(\theta) \cos(\theta) - a]$  is the matrix of kinematic constraint coefficients satisfying:  $A(q)\dot{q} = 0$  (10)

 $+ F = [F_x \quad F_y \quad T]$  is the general force acting on the system

- $F_{\chi} = \frac{1}{r}(\tau_R + \tau_L)\cos(\theta)$ : force in the x-axis direction  $T = \frac{b}{r}(\tau_R \tau_L)$ : the torque on the robot
- $F_y = \frac{1}{r}(\tau_R + \tau_L)sin(\theta)$ : force in the y-axis direction
- $\tau_R$ ,  $\tau_L$ : torques applied to the right and left wheels



#### B. WMR system model:

- 2. Dynamic model
- In case of no disturbance

According to Lagrange formulation (7), the following differential equations are obtained:

$$\begin{cases} m_G \ddot{x}_G = F_x + \lambda_1 \sin(\theta) \\ m_G \ddot{y}_G = F_y - \lambda_2 \cos(\theta) \\ I_G \ddot{\theta} = T + \lambda_3 a \end{cases}$$
 (11)

Or written in matrix form as:

$$\overline{M}\ddot{q} = \overline{E}\tau - A^T\lambda \tag{12}$$

Where,

$$\tau = [\tau_R \quad \tau_L]^T, \quad \overline{M} = \begin{bmatrix} m_G & 0 & 0 \\ 0 & m_G & 0 \\ 0 & 0 & I_G \end{bmatrix}, \quad \overline{E} = \frac{1}{r} \begin{bmatrix} \cos(\theta) & \cos(\theta) \\ \sin(\theta) & \sin(\theta) \\ b & -b \end{bmatrix}$$

#### B. WMR system model:

- 2. Dynamic model
- In case of no disturbance

$$\overline{M}\ddot{q} = \overline{E}\tau - A^T\lambda \tag{12}$$

The kinematics of the WMR (2) can be written in matrix form as:  $\dot{q} = S_0 v_0$  (13)

Where 
$$S_0 = \begin{bmatrix} \cos(\theta) & -a\sin(\theta) \\ \sin(\theta) & a\cos(\theta) \\ 0 & 1 \end{bmatrix}$$
,  $v_0 = [\Theta \quad \Omega]^T$ 

Taking the time derivative of (13):

$$\ddot{q} = \dot{S}_0 v_0 + S_0 \dot{v}_0 \tag{14}$$

Substitute (14) into (12): 
$$\overline{M}\dot{S}_0v_0 + \overline{M}S_0\dot{v}_0 = \overline{E}\tau - A^T\lambda$$
 (15)

Multiply (15) by 
$$S^T$$
 gives:  $M\dot{v}_0 + \bar{B}v_0 = E\tau$  (16)

Where 
$$\mathbf{M} = \mathbf{S}^T \overline{\mathbf{M}} \mathbf{S} = \begin{bmatrix} m_G & 0 \\ 0 & m_G a^2 + I_G \end{bmatrix}$$
,  $\overline{B} = S^T \overline{M} \dot{S} = \begin{bmatrix} 0 & -m_G a \Omega \\ m_G a \Omega & 0 \end{bmatrix}$ ,  $E = S^T \overline{E} = \begin{bmatrix} \frac{1}{r} & \frac{1}{r} \\ \frac{b}{r} & -\frac{b}{r} \end{bmatrix}$ 



#### B. WMR system model:

- 2. Dynamic model
- In case of wheel slip and external disturbances

The Lagrangian is defined as:  $L = K - U = K_G + K_R + K_L$  (17)

$$K_G = \frac{1}{2} m_G (\dot{x}_G^2 + \dot{y}_G^2) + \frac{1}{2} I_G \omega^2 \qquad \text{: the kinetic energy of the WMR platform.}$$
 Where 
$$\begin{cases} K_R = \frac{1}{2} m_W \left[ \left( r \dot{\phi}_R + \dot{\gamma}_R \right)^2 + \eta^2 \right] + \frac{1}{2} I_W \dot{\phi}_R^2 + \frac{1}{2} I_D \omega^2 \text{: the kinetic energy of the right wheel.} \\ K_L = \frac{1}{2} m_W \left[ \left( r \dot{\phi}_R + \dot{\gamma}_L \right)^2 + \eta^2 \right] + \frac{1}{2} I_W \dot{\phi}_L^2 + \frac{1}{2} I_D \omega^2 \text{: the kinetic energy of the left wheel.} \end{cases}$$

- m<sub>W</sub>: mass of each wheel
- $I_W$ : the moment of inertia of each driving wheel about its rotational axis
- $I_D$ : the moment of inertia of each driving wheel about its diameter axis



#### B. WMR system model:

- 2. Dynamic model
- In case of wheel slip and external disturbances

Based on (4), the nonholonomic constraint equations can be written as:

$$\begin{cases} \dot{\gamma}_R = -r\dot{\phi}_R + \dot{x}_G\cos(\theta) + \dot{y}_G\sin(\theta) + b\omega \\ \dot{\gamma}_L = -r\dot{\phi}_L + \dot{x}_G\cos(\theta) + \dot{y}_G\sin(\theta) - b\omega \\ \dot{\eta} = -\dot{x}_G\sin(\theta) + \dot{y}_G\cos(\theta) \end{cases}$$
(18)

Similarly, the Lagrange formulation can be obtained as:  $\frac{d}{dt} \left( \frac{\partial L}{\partial \dot{q}} \right) - \frac{\partial L}{\partial q} + \bar{\tau}_d = N\tau - A^T\lambda$  (19)

Where  $\bar{\tau}_d$  illustrates model uncertainties and unknown disturbances, N is the input transformation matrix.

• 
$$\mathbf{q} = [x_G \quad y_G \quad \theta \quad \eta \quad \gamma_R \quad \gamma_L \quad \phi_R \quad \phi_L]^T$$
 is the general Lagrange coordinate

• 
$$A(q) = \begin{bmatrix} \cos(\theta) & \sin(\theta) & b & 0 & -1 & 0 & -r & 0 \\ \cos(\theta) & \sin(\theta) & -b & 0 & 0 & -1 & 0 & -r \\ -\sin(\theta) & \cos(\theta) & a & -1 & 0 & 0 & 0 \end{bmatrix}$$
 which satisfies  $A(q)\dot{q} = 0$ 



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#### B. WMR system model:

- 2. Dynamic model
- In case of wheel slip and external disturbances

Solve the Lagrange equation gives: 
$$\overline{M}\ddot{q} + \overline{\tau}_d = N\tau - A^T\lambda$$
 (20) where:

$$M = \begin{bmatrix} m_G & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & m_G & 0 & 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & I_G + 2I_D & 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 2m_W & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & m_W & 0 & m_W r & 0 \\ 0 & 0 & 0 & 0 & 0 & m_W r & 0 & m_W r \\ 0 & 0 & 0 & 0 & 0 & m_W r & 0 & m_W r^2 + I_W \end{bmatrix}, N = \begin{bmatrix} 0 & 0 \\ 0 & 0 \\ 0 & 0 \\ 0 & 0 \\ 0 & 0 \\ 1 & 0 \\ 0 & 1 \end{bmatrix}$$



#### B. WMR system model:

- 2. Dynamic model
- In case of wheel slip and external disturbances

The actual kinematics (6) can be written as:  $\dot{q} = S_1(q)v + S_2(q)\dot{\gamma} + S_3(q)\dot{\eta}$  (21) where:

$$\bullet \ S_1 = \begin{bmatrix} \frac{r\cos(\theta)}{2} - \frac{ar\sin(\theta)}{2b} & \frac{r\cos(\theta)}{2} + \frac{ar\sin(\theta)}{2b} \\ \frac{r\sin(\theta)}{2} + \frac{ar\cos(\theta)}{2b} & \frac{r\sin(\theta)}{2} - \frac{ar\cos(\theta)}{2b} \\ \frac{r}{2} & \frac{r}{2b} & -\frac{r}{2b} \\ 0 & 0 & 0 \\ 0 & 0 & 0 \\ 1 & 0 & 0 \\ 0 & 0 & 1 \end{bmatrix}, \ S_2 = \begin{bmatrix} \frac{\cos(\theta)}{2} - \frac{a\sin(\theta)}{2b} & \frac{\cos(\theta)}{2} - \frac{a\sin(\theta)}{2b} \\ \frac{\sin(\theta)}{2} + \frac{a\cos(\theta)}{2b} & \frac{\sin(\theta)}{2} - \frac{a\cos(\theta)}{2b} \\ \frac{\sin(\theta)}{2} + \frac{a\cos(\theta)}{2b} & \frac{\sin(\theta)}{2} - \frac{a\cos(\theta)}{2b} \\ \frac{1}{2b} & -\frac{1}{2b} & -\frac{1}{2b} \\ 0 & 0 & 0 \\ 1 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 0 \\ 0 & 0 & 0 \end{bmatrix}, \ S_3 = \begin{bmatrix} -\sin(\theta) \\ \cos(\theta) \\ \cos(\theta) \\ 0 \\ 1 \\ 0 \\ 0 \\ 0 \end{bmatrix}$$

• 
$$v = [\dot{\phi}_R \quad \dot{\phi}_L]^T$$
 ,  $\gamma = [\gamma_R \quad \gamma_L]^T$ 



#### B. WMR system model:

- 2. Dynamic model
- In case of wheel slip and external disturbances

$$\overline{M}\ddot{q} + \overline{\tau}_d = N\tau - A^T\lambda \qquad (20)$$

Taking the time derivative of (21): 
$$\ddot{q} = \dot{S}_1 v + S_1 \dot{v} + \dot{S}_2 \dot{\gamma} + S_2 \ddot{\gamma} + \dot{S}_3 \dot{\eta} + S_3 \ddot{\eta}$$
 (22)

Substitute (22) into (20) and multiply both sides by  $S_1^T$  gives the actual dynamic model:

$$M\dot{v} + Bv + Q\ddot{\gamma} + C\dot{\eta} + G\ddot{\eta} + \tau_d = \tau \tag{23}$$

Where 
$$\tau_d = S_1^T \bar{\tau}_d$$
;  $M = S_1^T \overline{M} S_1 = \begin{bmatrix} m_{11} & m_{12} \\ m_{21} & m_{22} \end{bmatrix}$ ;  $m_{22} = m_{11}$ ,  $m_{21} = m_{12}$ ;  $Q = S_1^T \overline{M} S_2 = \begin{bmatrix} Q_1 & Q_2 \\ Q_2 & Q_1 \end{bmatrix}$ ;

$$G = S_1^T \overline{M} S_3 = \frac{m_G a r}{2b} \begin{bmatrix} 1 \\ -1 \end{bmatrix}, B = S_1^T \overline{M} \dot{S}_1 = \frac{m_G a r^2}{2b} \omega \begin{bmatrix} 0 & 1 \\ -1 & 0 \end{bmatrix},$$

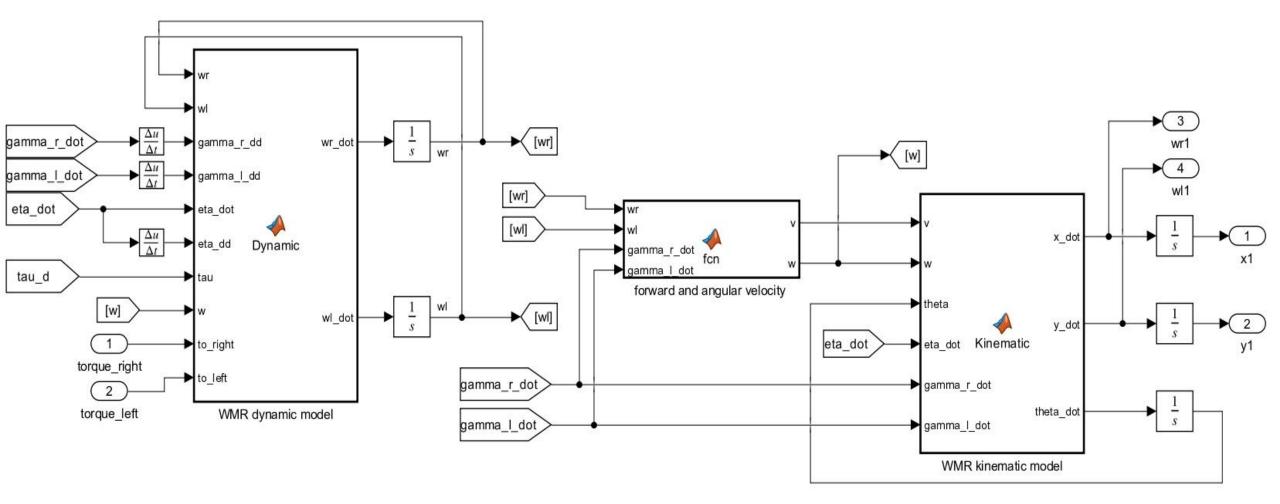
$$(r^2 - a^2 r^2) = r^2$$

$$m_{11} = m_G \left(\frac{r^2}{4} + \frac{a^2 r^2}{4b^2}\right) + \frac{r^2}{4b^2} (I_G + 2I_D) + m_W r^2 + I_W , m_{12} = m_G \left(\frac{r^2}{4} - \frac{a^2 r^2}{4b^2}\right) - \frac{r^2}{4b^2} (I_G + 2I_D)$$

$$Q_{1,2} = m_G \frac{r}{4} \left( 1 \pm \frac{a^2}{b^2} \right) \pm \frac{r}{4b} (I_G + 2I_D)$$



#### C. WMR simulation implementation:



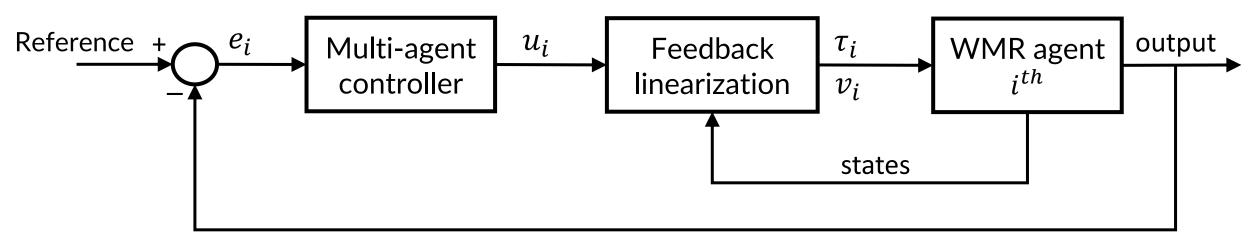




## Multi-agent control design using traditional method

#### A. Control objective:

- The consensus algorithm is utilized for agents with single-integrator and double-integrator dynamics to achieve common information states.
- Distributed control law is implemented for each agent to ensure formation tracking while accurately following predefined trajectories.
- → Apply the control algorithm to the WMR system.

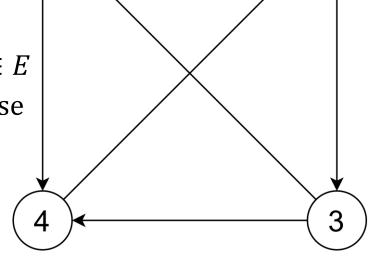


#### A. Control objective:

- Each agent interacts and exchanges relevant information with other agents through the communication graph.

- Consider a network of n agents with graph  $G = \{V, E\}$  where:
- $V = \{1, 2, ..., n\}$  is a set of nodes
- $E \subset V \times V$  is a set of edges
- The adjacency matrix is  $A = \begin{bmatrix} a_{ij} \end{bmatrix}_{n \times n} \in \mathbb{R}^{n \times n}$ ,  $\begin{cases} a_{ij} = 1 \ if \ (i,j) \in E \\ a_{ij} = 0 \end{cases}$ , otherwise
- The degree matrix is  $D = diag\{d_1, ..., d_n\} \in \mathbb{R}^{n \times n}$ ,  $d_i = \sum_{j=1}^n a_{ij}$
- The Laplacian matrix is  $\mathcal{L} = D A = \left[l_{ij}\right]_{n \times n} \in \mathbb{R}^{n \times n}$ , where:

$$l_{ij} = \begin{cases} -a_{ij} = 1, & \text{if } i, j \in V, i \neq j \\ \sum_{j=1}^{n} a_{ij}, & \text{if } i = j \end{cases}$$



Interaction graph of 4 agents

#### B. Consensus algorithm for single-integrator models:

Consider the case where each agent is modeled as single-integrator dynamics, expressed as:

$$\dot{p}_{ri} = u_i \quad (24)$$

where  $p_{ri}$ , and  $u_i$  are the position and the control input of agent  $i^{th}$ , respectively.

The control law implemented for each agent:

$$u_{i} = \dot{p}_{ri}^{d} - k_{p} (p_{ri} - p_{ri}^{d}) + \sum_{j \in N_{i}} a_{ij} ((p_{rj} - p_{rj}^{d}) - (p_{ri} - p_{ri}^{d})), i = 1, ..., n \quad (25)$$

- $N_i = \{j \in V | (i, j) \in E\}$ : the set of neighborhood of agents  $i^{th}$   $p_{ri}^d = \begin{bmatrix} x_{ri}^d & y_{ri}^d \end{bmatrix}^T$ : desired position
- $p_{ri} = [x_{ri} \quad y_{ri}]^T$ : actual position  $u_i = [u_{xi} \quad u_{yi}]^T$ : control input

Let  $p_r = [p_{r1} \quad p_{r2} \quad ... \quad p_{rn}]^T$ ,  $p_r^d = [p_{r1}^d \quad p_{r2}^d \quad ... \quad p_{rn}^d]^T$ , the control law is rewritten as:

$$u = -\left(\left(k_p I_n + \mathcal{L}\right) \otimes I_2\right) \left(p_r - p_r^d\right) \tag{26}$$

Where  $u = [u_1 \quad u_2 \quad ... \quad u_n]^T$ ,  $I_n$  is the identity matrix of size n



#### B. Consensus algorithm for single-integrator models:

The positions of each WMR agent under no disturbance (13) can be rewritten as:

$$\dot{p}_i = S v_i \quad (27)$$
 Where  $S = \begin{bmatrix} \cos(\theta) & -a\sin(\theta) \\ \sin(\theta) & a\cos(\theta) \end{bmatrix}$ ,  $v_i = [\Theta_i \quad \Omega_i]^T$ 

Feedback linearization:

$$v_i = \begin{bmatrix} \cos(\theta) & \sin(\theta) \\ -\frac{1}{a}\sin(\theta) & \frac{1}{a}\cos(\theta) \end{bmatrix} u_i \quad (28), \text{ where } u_i = \begin{bmatrix} \mathbf{u}_{xi} & u_{yi} \end{bmatrix}^T$$

The model becomes:

$$\dot{p}_i = u_i \tag{29}$$

→ Control law (26) can be directly applied.

#### C. Consensus algorithm for double-integrator models:

In case the dynamics of each agent follows double-integrator dynamics, defined as:

$$\begin{cases} \dot{p}_{ri} = v_{ri} \\ \dot{v}_{ri} = u_i \end{cases} \tag{30}$$

where  $p_{ri}$ ,  $v_{ri}$  and  $u_i$  are the position, velocity and the control input of agent  $i^{th}$ , respectively.

The control law implemented for each agent:

$$u_{i} = -k_{p}(p_{ri} - p_{ri}^{d}) + \sum_{j \in N_{i}} a_{ij} \left( (p_{rj} - p_{rj}^{d}) - (p_{ri} - p_{ri}^{d}) \right) - k_{v}(v_{ri} - \dot{p}_{ri}^{d}), i = 1, \dots, n \quad (31)$$

Or rewritten as:

$$u = -\left(\left(k_p I_n + \mathcal{L}\right) \otimes I_2\right) \left(p_r - p_r^d\right) - k_v \left(v_r - \dot{p}_r^d\right) \tag{32}$$



#### C. Consensus algorithm for double-integrator models:

The dynamic model of each WMR agent under no disturbance (16) can be rewritten as:

$$\dot{v}_i = Bv_i + N\tau_i \quad (33)$$

$$\mathbf{B} = -\mathbf{M}^{-1} \overline{\mathbf{B}} = \begin{bmatrix} 0 & a\Omega_{\mathrm{i}} \\ \frac{-m_{G}a\Omega}{I_{G} + m_{G}a} & 0 \end{bmatrix}, \ N = M^{-1}E = \begin{bmatrix} \frac{1}{m_{G}r} & \frac{1}{m_{G}r} \\ \frac{b}{r(I_{G} + m_{G}a^{2})} & \frac{-b}{r(I_{G} + m_{G}a^{2})} \end{bmatrix}, \ v_{i} = \begin{bmatrix} \Theta_{i} \\ \Omega_{i} \end{bmatrix}, \ \tau_{i} = \begin{bmatrix} \tau_{Ri} \\ \tau_{Li} \end{bmatrix}$$

And the kinematic equations (27) obtained as:  $\dot{p}_i = Sv_i$ . Define:  $z_i = Sv_i$  (34)

Derivative of  $z_i$ :

$$\dot{z}_i = \dot{S}v_i + S\dot{v}_i \tag{35}$$

$$\dot{z}_i = (\dot{S} + SB)v_i + SN\tau_i \tag{36}$$

$$\tau_i = N^{-1} S^{-1} (u_i - (\dot{S} + SB) v_i) \tag{37}$$

$$\begin{cases} \dot{p}_i = z_i \\ \dot{z}_i = u_i \end{cases} \tag{38}$$



→ Control law (32) can be directly applied.

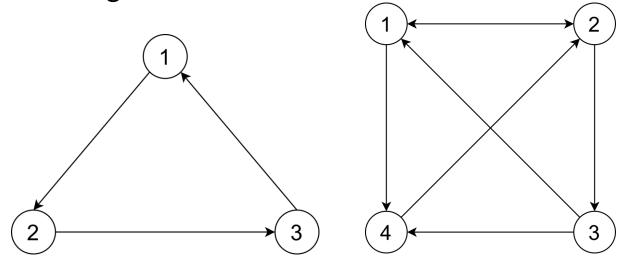


# Multi-agent control design using Adaptive Dynamic Programming approach

#### A. Control objective:

- An adaptive optimal controller in cooperation with a disturbance observer is proposed to ensure formation tracking for WMR agents in case of wheel slip and external disturbances.
- A multi-robot system with n WMRs achieve formation geometry of square and triangle shape, following two types of trajectories: line and circle.

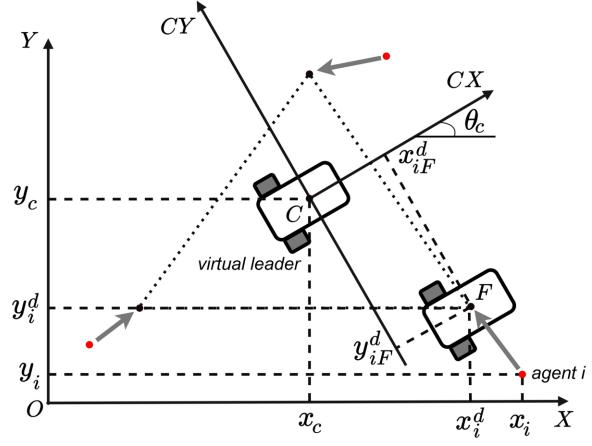
- For comparison with previous approach, the WMR formation is tasked to follow a curved trajectory under square configuration.





#### **B. Problem formulation:**

- Virtual leader/structure approach.
- Decentralized scheme instead of a centralized one.
- Control approach:
- + Define the desired dynamics of the virtual leader or virtual structure.
- + The motion of the virtual leader is translated into desired motions for each robot.
- + Derive distributed tracking control laws for each robot.



Triangle formation with a known virtual leader



#### **B. Problem formulation:**

The formation structure of the WMR system is derived as:

$$p_i^d = p_c + \begin{bmatrix} \cos(\theta_c) & -\sin(\theta_c) \\ \sin(\theta_c) & \cos(\theta_c) \end{bmatrix} p_{iF}^d \quad (39)$$

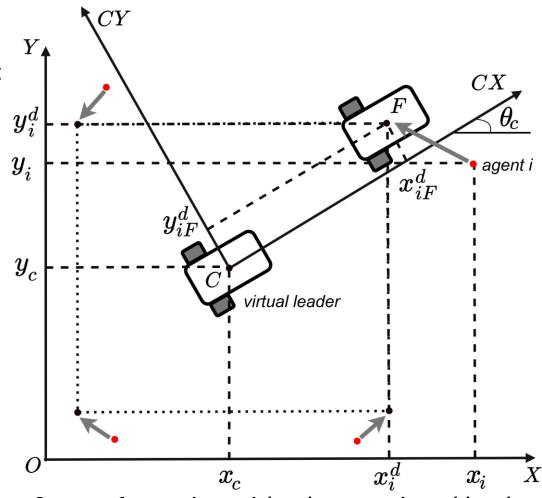
- $p_i^d = \begin{bmatrix} x_i^d & y_i^d \end{bmatrix}^T$ : desired position
- $p_c = [x_c \ y_c]^T$ : position of virtual leader
- $p_{iF}^d = \begin{bmatrix} x_{iF}^d & y_{iF}^d \end{bmatrix}^T$ : desired deviation of the i<sup>th</sup> robot

Where 
$$x_{iF}^d = l_i \cos(\phi_i)$$
 ,  $y_{iF}^d = l_i \sin(\phi_i)$ 

- +  $l_i$ : the distance from the formation center to each agent
- +  $\phi_i$  : is the angular offsets for each agent

$$\phi_i = \frac{\pi}{2}(i-1)$$
 ,  $i=1,...,4$  for square shape

$$\phi_i = \frac{2\pi}{3}(i-1)$$
,  $i = 1, ..., 3$  for triangle shape



Square formation with a known virtual leader



→ Control law (32) using consensus algorithm can be applied.

#### C. Tracking control design:

#### 1. Control strategy

The consensus-based control law requires each WMR to approximate the double-integrator dynamics (30):

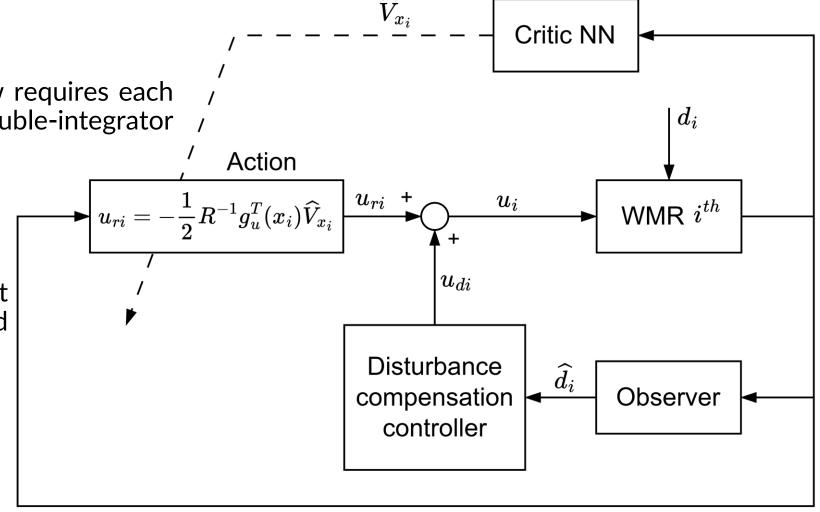
$$\begin{cases} \dot{p}_{ri} = v_{ri} \\ \dot{v}_{ri} = u_i \end{cases}$$

The kinematics of the  $i^{th}$  robot under disturbance is obtained from (5):

$$\dot{p}_i = Sv_i \quad (40)$$

Define the tracking error:

$$e_i = p_i - p_{ri} \quad (41)$$







#### C. Tracking control design:

#### 1. Control strategy

The error dynamics is derived as:

$$\dot{e}_i = \dot{p}_i - \dot{p}_{ri} = Sv_i + d_p - v_{ri}$$
 (42)

The time derivative of (42):

$$\ddot{e}_i = \dot{S}v_i + S\dot{v}_i + \dot{d}_p - u_i \tag{43}$$

The dynamic model for  $i^{th}$  robot (23) can be rewritten as:

$$\dot{v}_i = -M^{-1}Bv_i + M^{-1}\tau_i + d_v \quad (44) \ \text{, where} \quad d_v = -M^{-1}(Q\ddot{\gamma} + C\dot{\eta} + G\ddot{\eta} + \tau_d)$$

Substitute (44) into (43):  $\ddot{e}_i = -SM^{-1}Bv_i + u_i^{new} + \dot{S}v_i + Sd_v + \dot{d}_p$  (45)

Where  $u_i^{new} = SM^{-1}\tau_i - u_i$  is the transformed control input.

The control torque for the  $i^{th}$  WMR is calculated as:

$$\tau_i = MS^{-1}(u_i^{new} + u_i)$$
 (46)



#### C. Tracking control design:

#### 1. Control strategy

Define 
$$x_{1i}=e_i$$
,  $x_{2i}=\dot{x}_{1i}+\lambda x_{1i}$ , the time derivative of  $x_{1i}$  and  $x_{2i}$  is obtained as: 
$$\begin{cases} \dot{x}_{1i}=\dot{e}_i=Sv_i+d_p-v_{ri}\\ \dot{x}_{2i}=Ex_{2i}-E\lambda x_{1i}+u_i^{new}+d_i \end{cases} \tag{47}$$

Where: 
$$E=E_1S^{-1}$$
 ,  $E_1=-SM^{-1}B$  ,  $d_i=\xi_{2i}-E\xi_{1i}$  , 
$$\xi_{1i}=d_p-v_{ri}$$
 ,  $\xi_{2i}=Sd_v+\dot{S}v_i+\dot{d}_p+\lambda Sv_i+\lambda \xi_{1i}$ 

Rewrite (47) in state space form:  $\dot{x}_i = f(x_i) + g_u u_i^{new} + g_d d_i$  (48)

Where 
$$x_i = \begin{bmatrix} x_{1i} \\ x_{2i} \end{bmatrix}$$
 ,  $f(x_i) = \begin{bmatrix} x_{2i} - \lambda x_{1i} \\ E x_{2i} - \lambda E x_{1i} \end{bmatrix}$  ,  $g_u = \begin{bmatrix} 0 \\ 1 \end{bmatrix}$  ,  $g_d = \begin{bmatrix} 0 \\ 1 \end{bmatrix}$ 

The control input is derived as follows to achieve optimal performance:

$$u_i^{new} = u_{ri}(x_i) + u_{di}(x_i)$$
 (49)



#### C. Tracking control design:

2. Observer-based controller design

A disturbance observer is used to estimate disturbance  $d_i$  as follows:

$$\begin{cases}
\hat{d}_i = z_i + \rho(x_i) \\
\dot{z}_i = -h(x_i) \{g_d(x_i)[z_i + \rho(x_i)] + f(x_i) + g_u(x_i)u_i^{new} \}
\end{cases} (50)$$

#### Where:

- $\hat{d}_i$ : the estimation of  $d_i$
- $z_i$ : the internal state of the observer
- $\rho(x_i)$ : the nonlinear function vector to be designed
- $h(x_i)$ : the gain of the observer, defined by  $h(x_i) = \frac{\partial \rho(x_i)}{\partial x_i}$

The compensation control component is derived as:

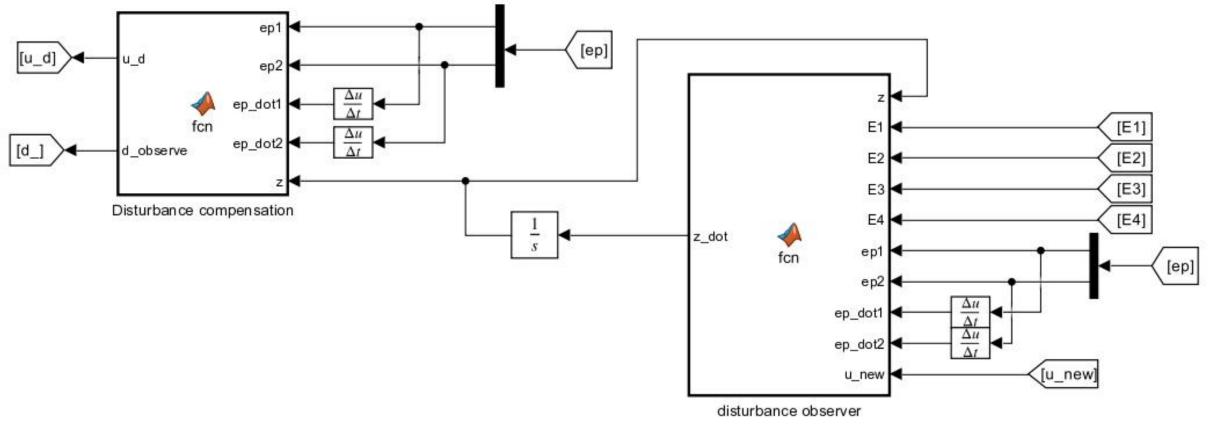
$$u_{di}(x_i) = -\hat{d}_i \qquad (51)$$



#### C. Tracking control design:

2. Observer-based controller design

The observer-based controller simulation implementation:



#### C. Tracking control design:

3. Adaptive optimal controller design

In the case of  $d_i = 0$  the error system (48) is rewritten as:

$$\dot{x}_i = f(x_i) + g_u u_i^{new} \tag{52}$$

Define the following cost function:  $_{\infty}$ 

$$J(x_i) = \int_t r\left(x_i(\tau_i), u_{ri}(x_i(\tau_i))\right) d\tau_i$$
 (53)

Where the utility function is defined as:  $r = x_i^T(\tau_i)Qx_i(\tau_i) + u_{ri}^T(x_i)Ru_{ri}(x_i)$ 

Define the Hamiltonian function of the problem as:

$$H = \left(\frac{\partial J}{\partial x_i}\right)^T + r(x_i, u_{ri}) = \left(\frac{\partial J}{\partial x_i}\right)^T + x_i^T Q x_i + u_{ri}^T R u_{ri}$$
 (54)

#### C. Tracking control design:

3. Adaptive optimal controller design

For the system (52) to have the optimal solution, there must exist an optimal cost function  $V(x_i, u_{ri})$  defined by:

$$V(x_i, u_{ri}) = \min_{u_{ri}} (J(x_i))$$
 (55)

Which satisfies the HJB equation:

$$0 = \min_{u_{ri}} \left( H(x_i, u_{ri}, V) \right) = \left( \frac{\partial V}{\partial x_i} \right)^T \dot{x}_i + x_i^T Q x_i + u_{ri}^T R u_{ri}$$
 (56)

The optimal control component  $u_{ri}$  can be derived as:

$$u_{ri} = \underset{u_{ri}}{\operatorname{argmin}} \left( H(x_i, u_{ri}, V) \right) \tag{57}$$

Solve (57) using (56) obtains:

$$u_{ri} = -\frac{1}{2}R^{-1}g_u^T \frac{\partial V}{\partial x_i}$$
 (58)



#### C. Tracking control design:

#### 3. Adaptive optimal controller design

The value function  $V(x_i, u_{ri})$  is approximated by a NN as:

$$V(x_i, u_{ri}) = W_i^T \phi(x_i) + \epsilon(x_i)$$
 (59)

The estimated cost function:

$$\widehat{V}(x_i, u_{ri}) = \widehat{W}_i^T \phi(x_i)$$
 (60)

The approximated Hamiltonian function:

$$H(x_i, u_{ri}, \widehat{W}_i) = \widehat{W}_i^T \left(\frac{\partial \phi}{\partial x_i}\right) \dot{x}_i + x_i^T Q x_i + u_{ri}^T R u_{ri} = e_i \quad (61)$$

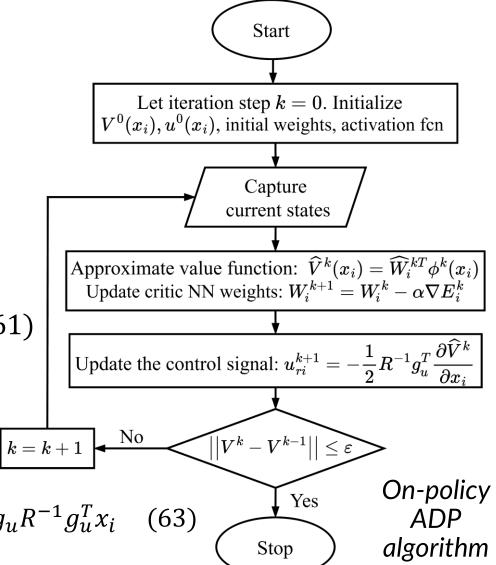
Update  $\widehat{W}_i$  to minimize the squared residual error:

$$E_i = \frac{1}{2} e_i^T e_i \quad (62)$$

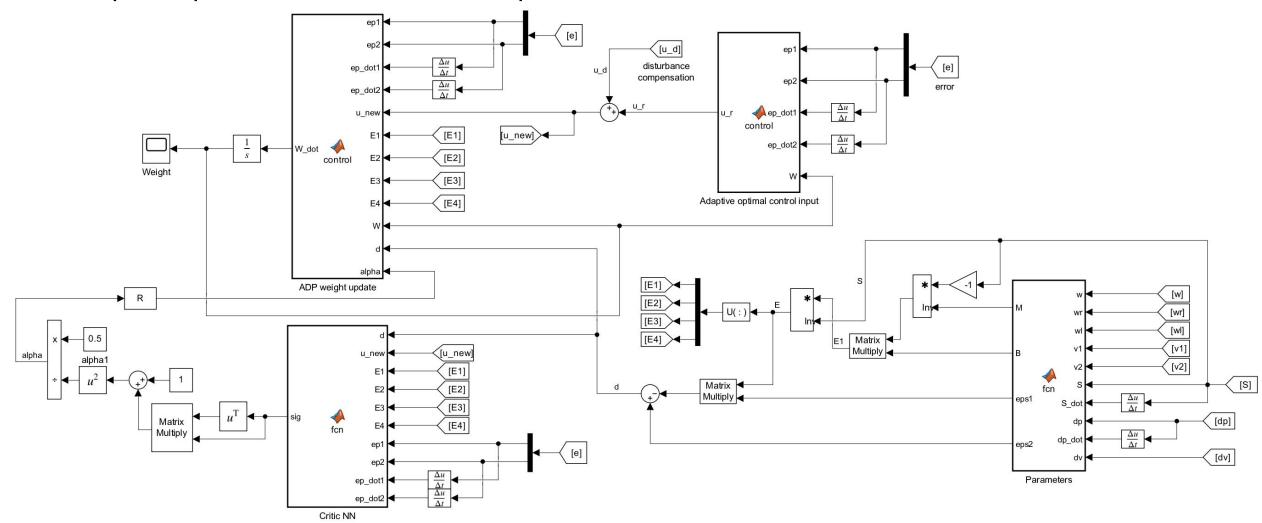
The weight update law is given by:

$$\widehat{W}_{i} = -\alpha_{1} \frac{\widehat{\sigma}_{i}}{\left(\widehat{\sigma}_{i}^{T} \widehat{\sigma}_{i} + 1\right)^{2}} \left(\widehat{\sigma}_{i}^{T} \widehat{W}_{i} + Q(x_{i}) + u_{ri}^{T} R u_{ri}\right) + \frac{1}{2} \alpha_{2} \frac{\partial \phi}{\partial x_{i}} g_{u} R^{-1} g_{u}^{T} x_{i}$$

$$\text{Where } \widehat{\sigma}_{i} = (\partial \phi / \partial x_{i}) \dot{x}_{i}$$



The adaptive optimal control simulation implementation:

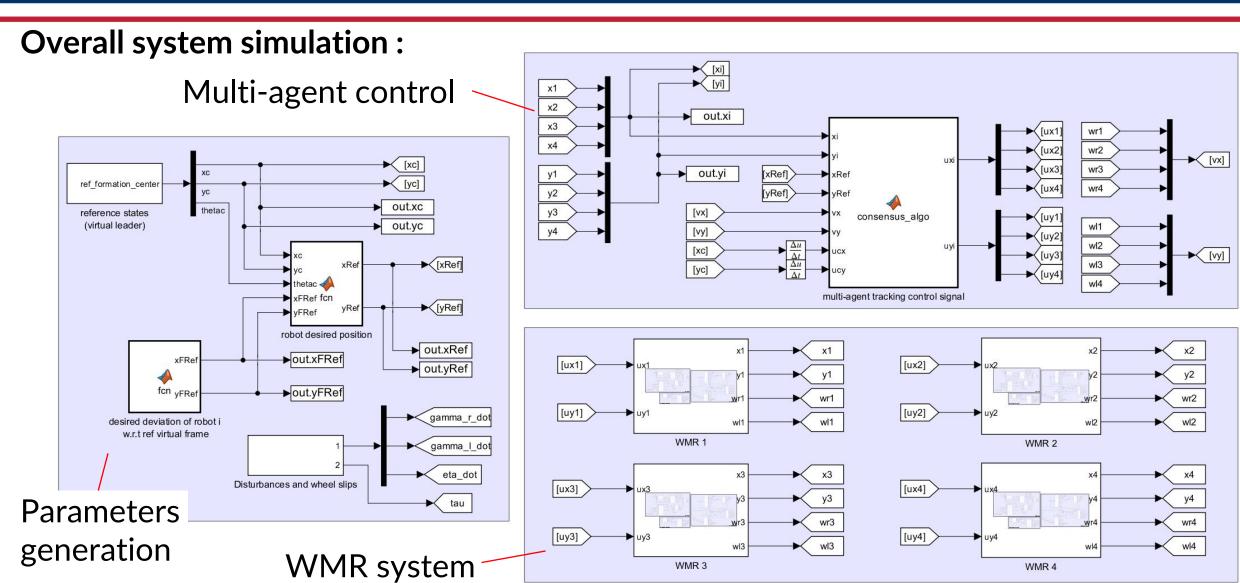




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### Simulation and Results

#### 5. Simulation implementation

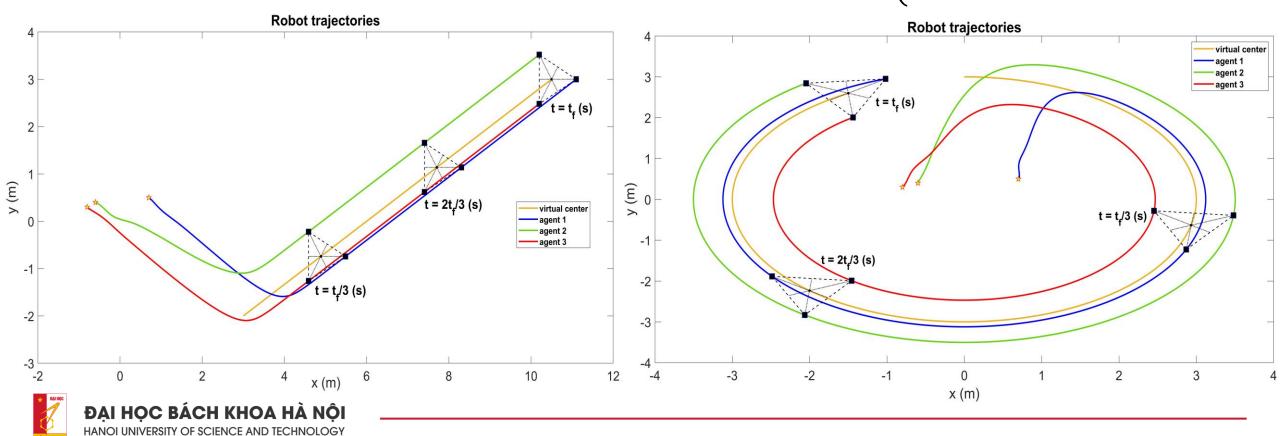


#### A. Formation tracking results of multiple WMRs

- Scenario 1: Triangle formation

Line trajectory: 
$$\begin{cases} x_c = 3 + 0.75t \\ y_c = -2 + 0.5t \end{cases}$$

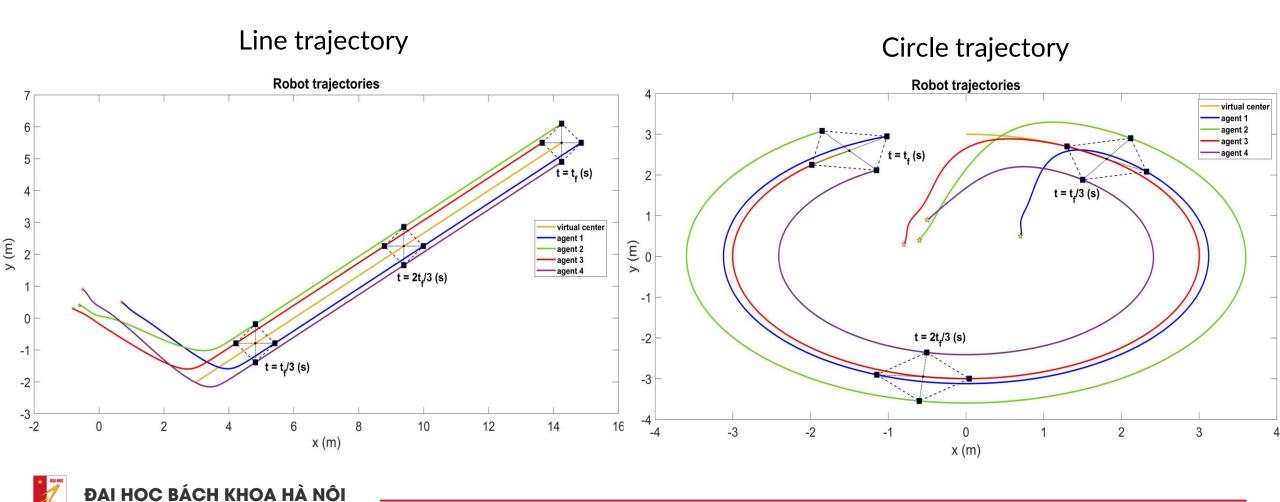
Circle trajectory:  $\begin{cases} x_c = 3\sin\left(\frac{\pi}{6}t\right) \\ y_c = 3\cos\left(\frac{\pi}{6}t\right) \end{cases}$ 



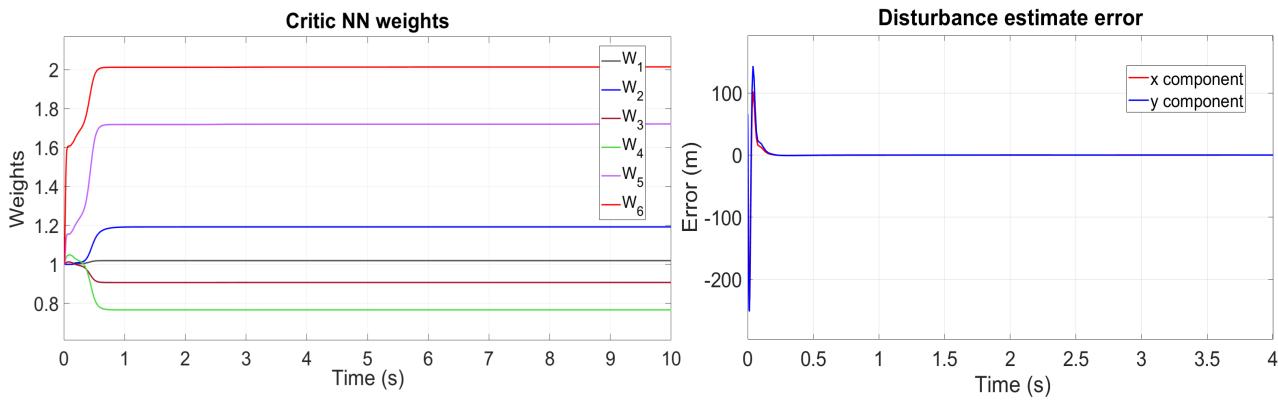
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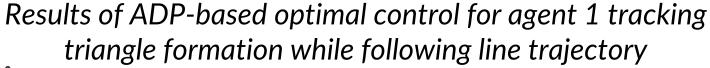
#### A. Formation tracking results of multiple WMRs

- Scenario 2: Square formation



#### A. Formation tracking results of multiple WMRs





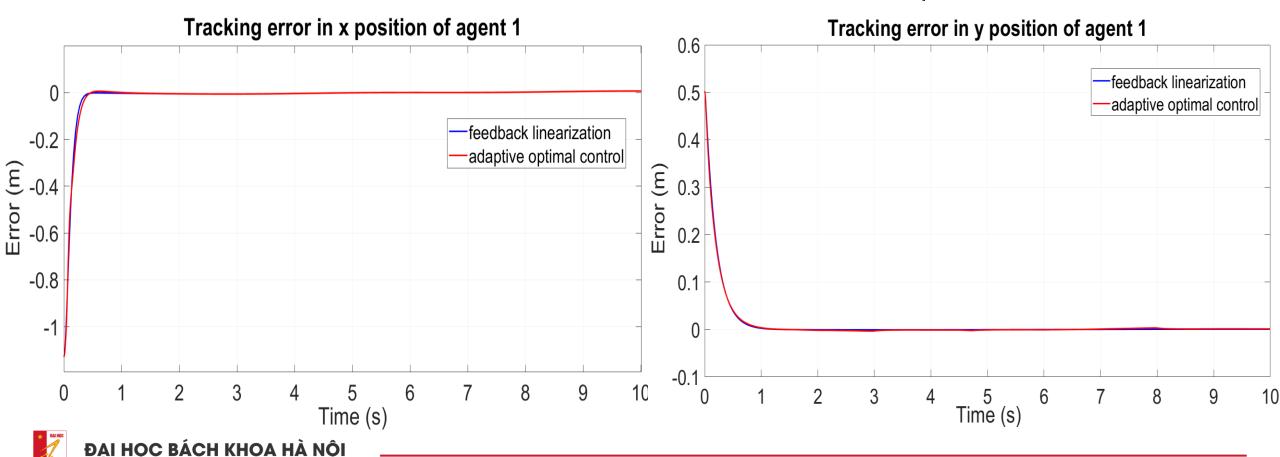


B. Comparison between ADP-based optimal control method and conventional feedback linearization control method:

In case of no disturbance:

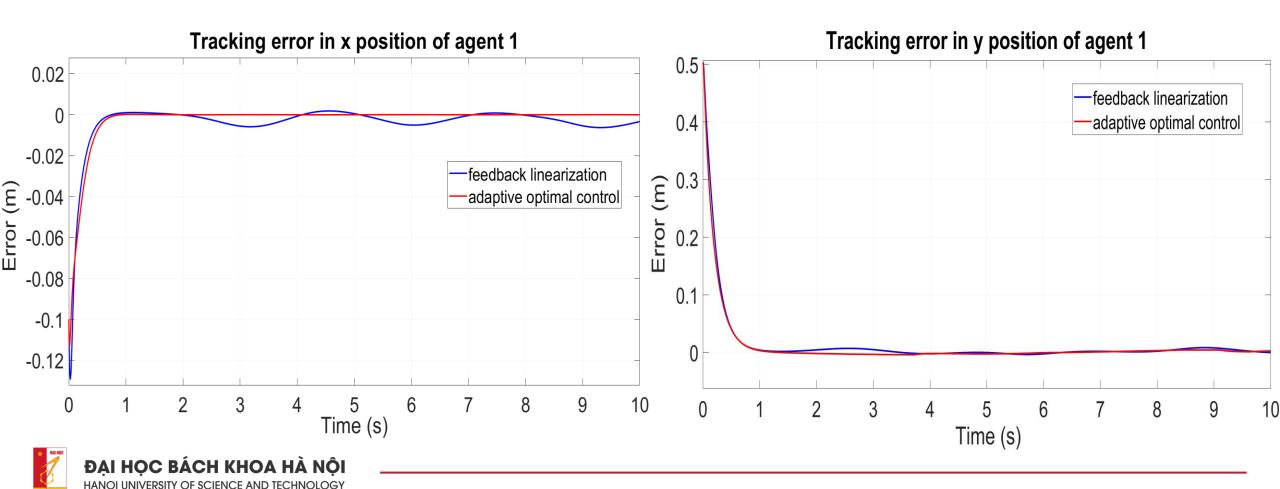
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Curved trajectory:  $\begin{cases} x_c = -t \\ y_c = \sin(0.5t) + 0.5t + 1 \end{cases}$ 



### B. Comparison between ADP-based optimal control method and conventional feedback linearization control method:

In case of wheel slip and external disturbance:



47

#### 6. Conclusion

- In general, the WMR system achieves the desired formations while following the predefined trajectories.
- The approach assumes each agent has consistent understanding of the formation center. Future study can focus on designing consensus algorithm to drive each agent's instantiation of the virtual center to a common value.
- The proposed method requires knowledge of the system's internal dynamics. Future improvement may focus on developing model-free control algorithms that rely solely on system state data.
- Possibly develop more effective multi-agent control algorithms. These advancements may include distance-based control, direction vector-based control, and graph rigidity theory-based methods.

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## THANK YOU!