

### **THEORY AND FLUENT DATA**

Volumetric flow rate	Q_V	8.33E-07	m^3/s
Width	W	0.04	m
Length	L	0.1	m
Gap	2H	0.0016	m
Aspect ratio	2H/W	0.04	1/AR=20
Area of cross section	A	0.000064	m^2
Flow velocity	U	0.013015625	m/s
Hydraulic diameter	DH	0.003076923	m
Temperature	T	288	K
Kinematic viscosity	$\nu$	1.00E-06	m2/s
Density	$\rho$	998	kg/m3
Specific heat	Cp	4,187	J/kg-K
Thermal conductivity	K	0.598	W/m-k
Reynolds number	Re	40.04807692	
Prandtl number	Pr	8.00	
Expected flow development length	x	0.012322485	m
Expected thermal development length	xt	0.02	m
Inlet temperature	T_in	288	K
Bulk temperature at exit	T_out	310	K
Q to be supplied for getting above temperature	Q	76.61	W
Surface area (each side)	As	0.004	m2
Heat flux on each surface for above Q	q	9576.25	W/m^2
Nu_analytical	Nu	8.325	
h_analytical	h	1617.96375	W/m^2-K
Temperature difference	dT	5.918704915	K
Estimated maximum wall temperature	T_wall_max	315.9187049	K
		42.76870492	C
Total heat transfer rate	inlet	-441.4024963	W
	outlet	58.35807800	W
	wall	383.04998	W

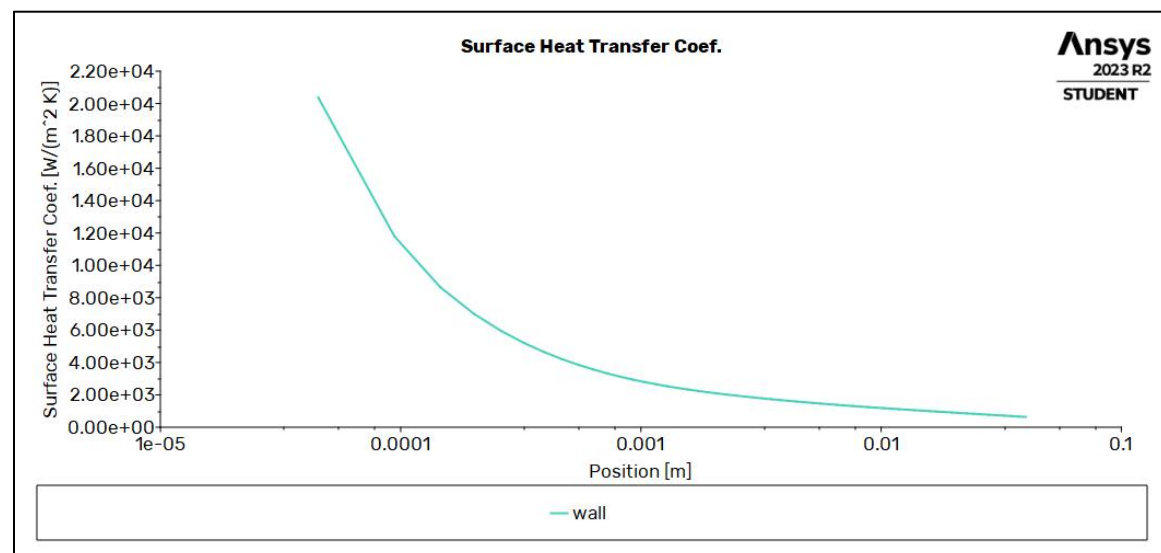
FLUENT			
Pressure drop	dP	2.575993	Pa
T_wall_max	T_wall_max	303	K
Wall shear	$\tau$	0.05112791	Pa
Friction coefficient	Cf	0.604821805	
Pumping power	Q_p	2.1458E-06	W
flow development length	x	0.0128	m
Nusselt No	Nu	8.42	

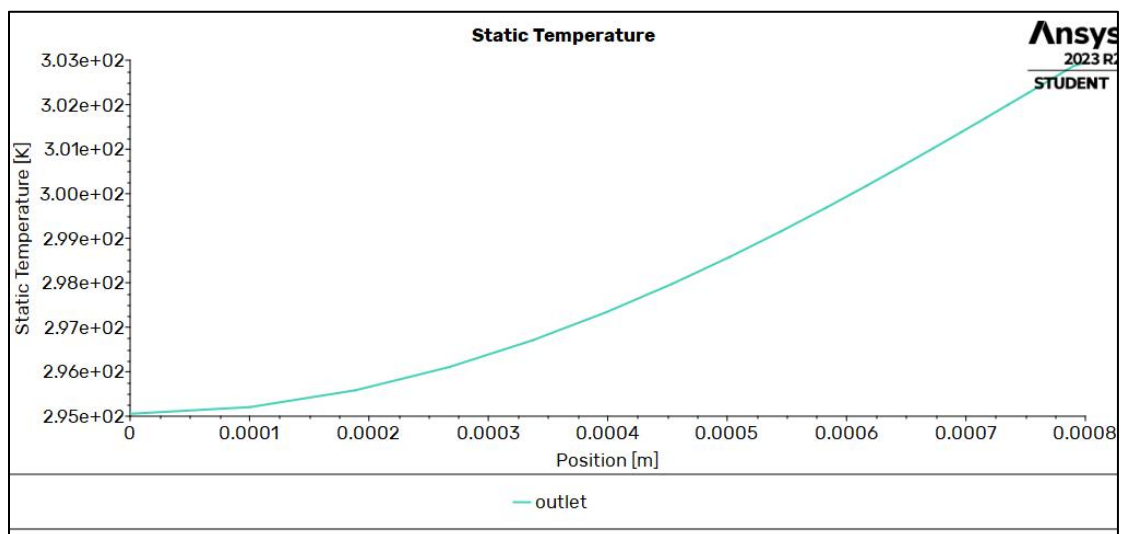
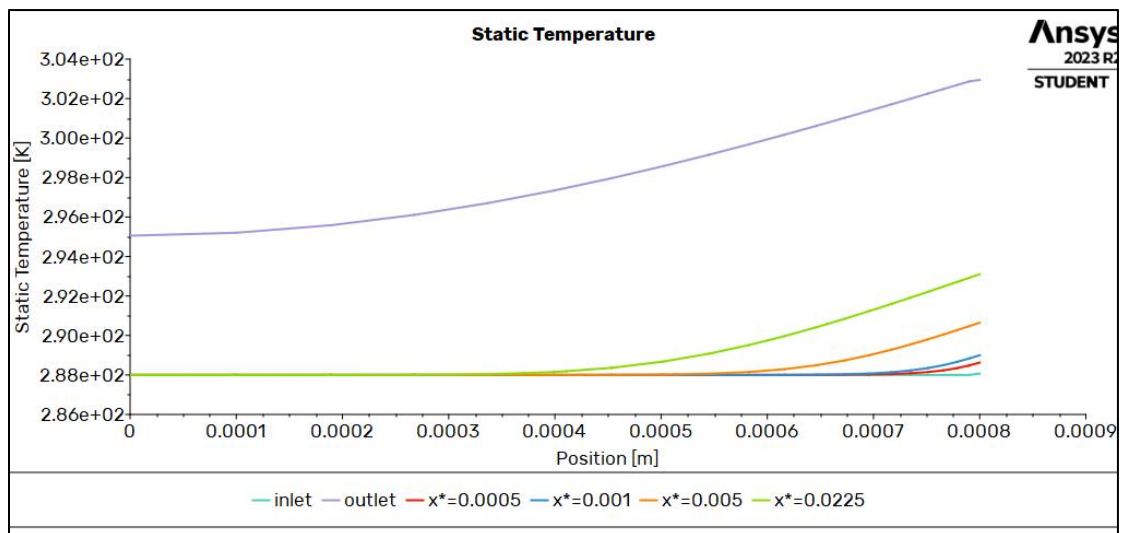
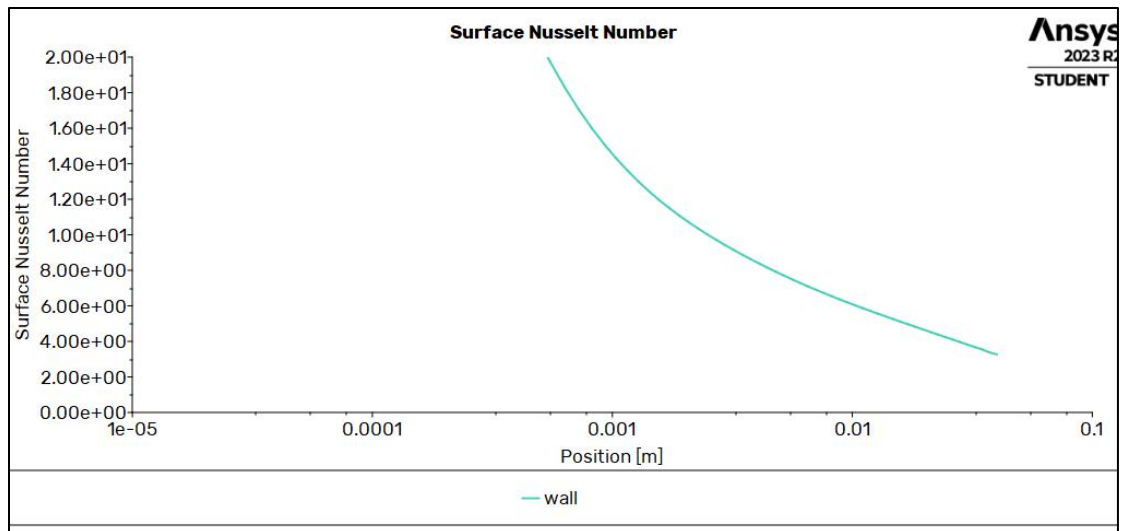
THEORY			
Wall shear for fully developed flow	$\tau$	0.050659416	Pa
Friction coefficient	Cf	0.599279712	
Pressure drop for the length L	dP	1.646431008	Pa
Pumping power	Q_p	1.37148E-06	W

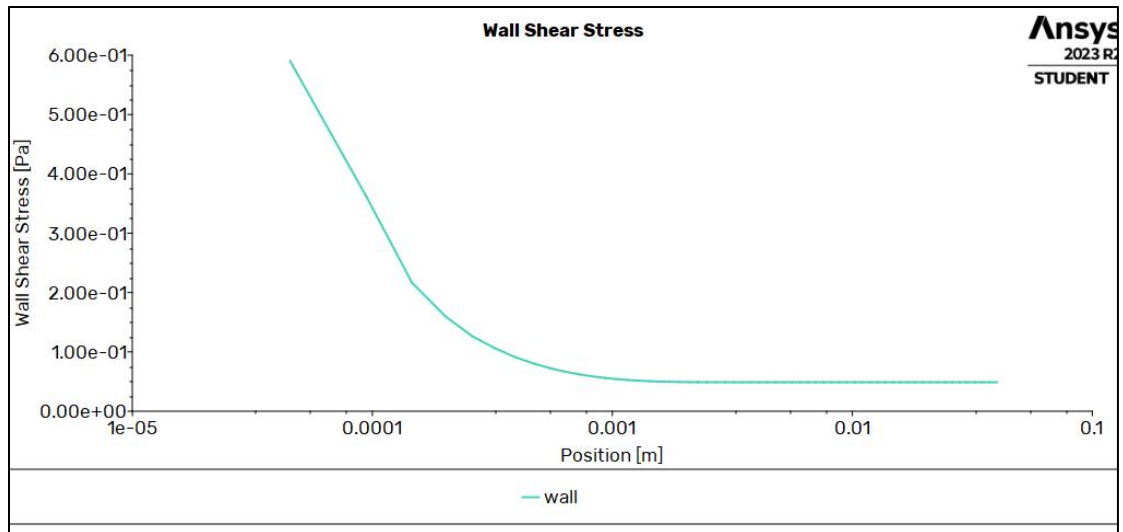
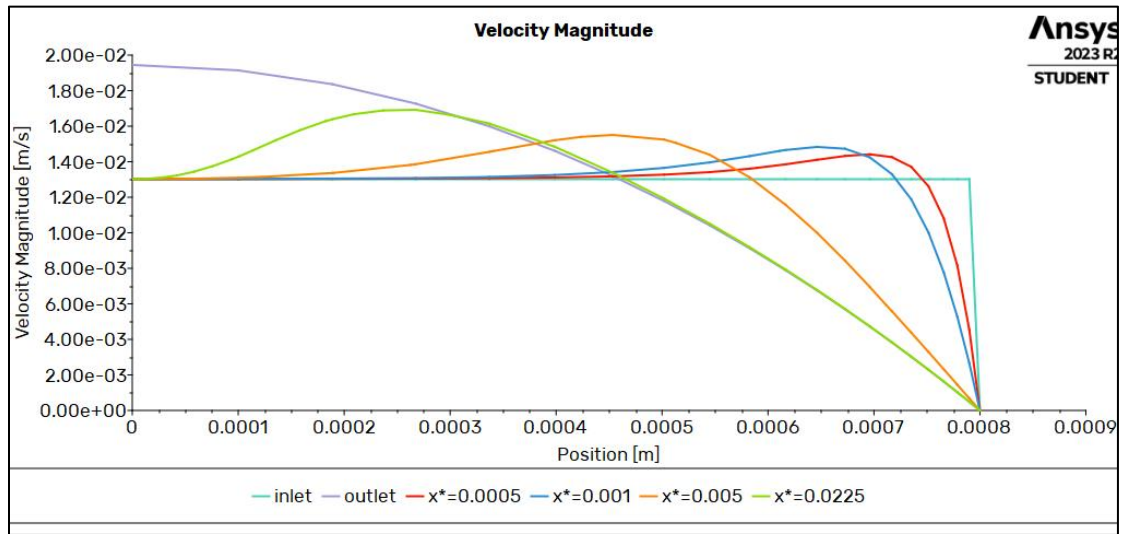
ERROR ESTIMATION			
Friction coefficient		0.9247923	%
Pumping power		56.45921316	%
Flow development length		3.87515006	%

#### FLUENT PLOT RESULTS

Sample plotting coordinates	x2	6.16124E-05	m
	x2	0.000123225	m
	x2	0.000616124	m
	x2	0.002772559	m
	y2	0.0008	m

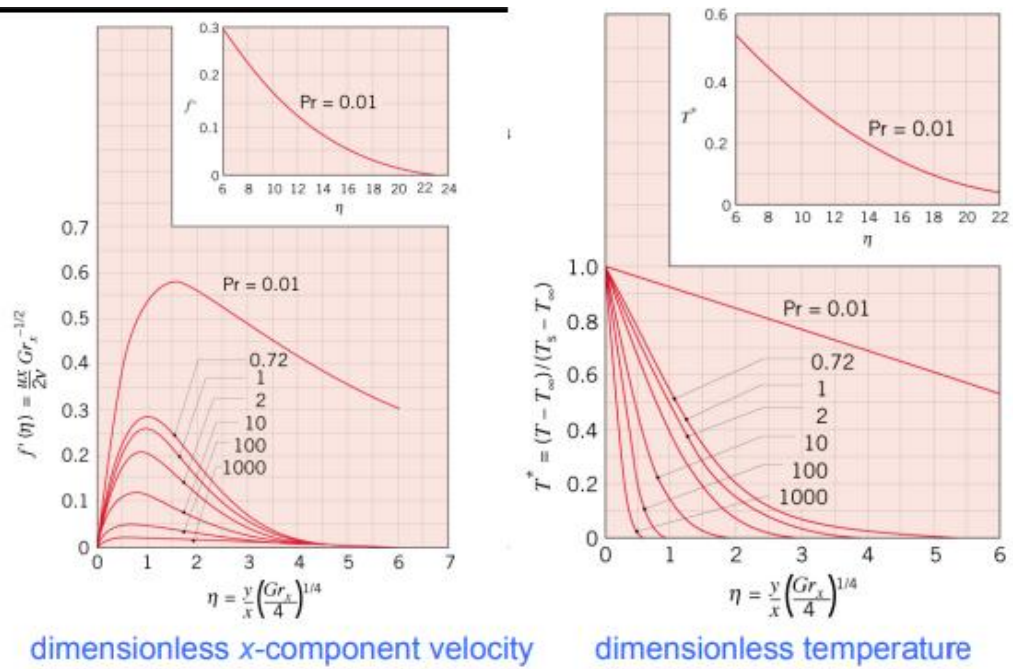






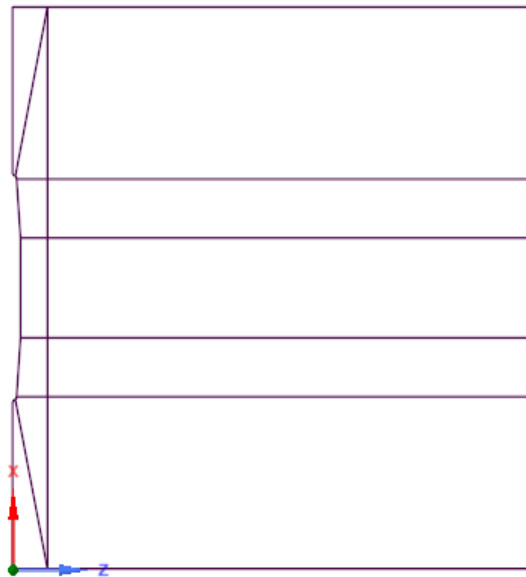
				<b>NATURAL CONVECTION</b>					
				length(L)	10	mm			
				Tw	78	deg C			
				Tambient	25	deg C			
				T_mean	51.5	deg C			
			<b>Properties of air at T_mean</b>						
				Nu					
				Pr	0.7228				
				nu	0.00001798	m^2/s			
				beta	0.00308166	K^-1			
	(g*beta*(Tw-Tamb)*L^3)/nu^2			Gr	4956.22257				
	Gr*Pr			Ra	3582.357673				
			<b>As Ra &lt;(10^9) ,Convection is laminar</b>						
			<b>Calculate BL thickness at TE ,by considering eta=6 from fig</b>						
				at eta	6				
eta=(y/x)*((Gr/4)^0.25))			Thermal BL thickness		0.010112966	m			
					10.11296602	mm			
			<b>From Fig the max.Velocity is at eta=1</b>						
				eta	1				
				f(eta)	0.28				
				T*	0.5				
	T*=(T-Tamb)/(Tw-Tamb)			T_max vel	324.5	K		<b>From CFD</b>	
				y_max vel	1.68	mm		1mm	
	f(eta)=(ux/2*nu)*(Gr^-0.5)			max_vel	0.071	m/s		0.11m/s	
		<b>FROM CFD</b>	<b>From emperical</b>						
	h (W/m^2-K)	12.11744	13.01						
	q (W/m^2)	642.2243	689.49						
	Q (W)	1.503677	1.614						

	% ERROR
y_max vel	40.5
max_vel	-54.9
h (surface heat transfer coefficient)	6
Q( heat rate)	7

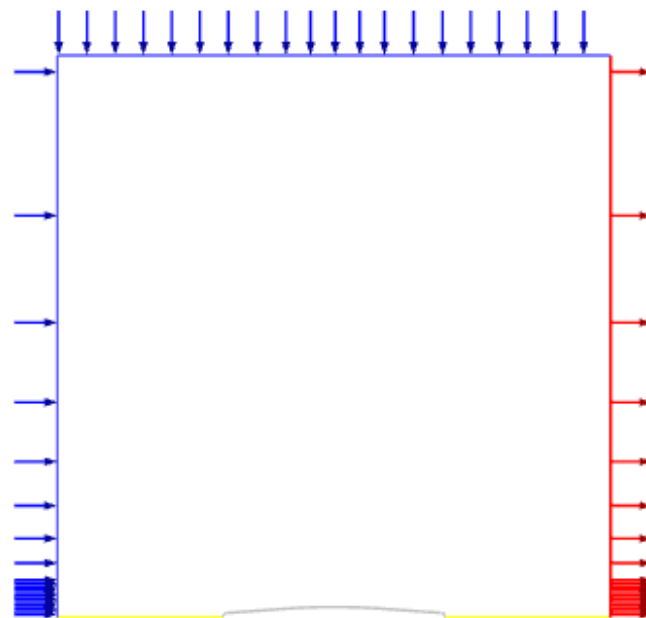
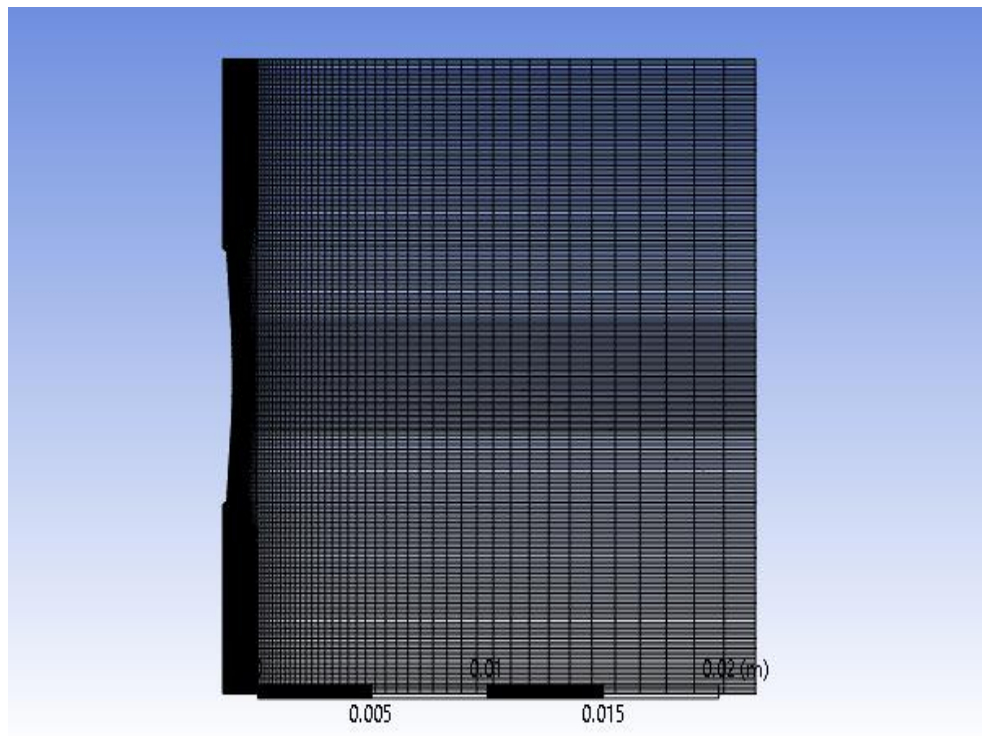


dimensionless x-component velocity

dimensionless temperature

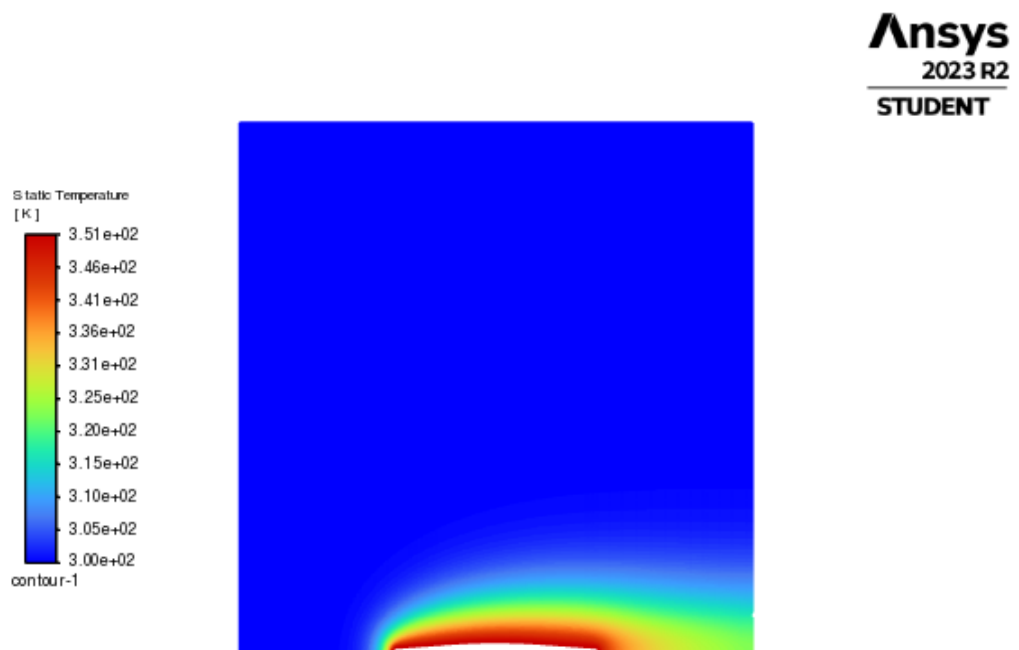
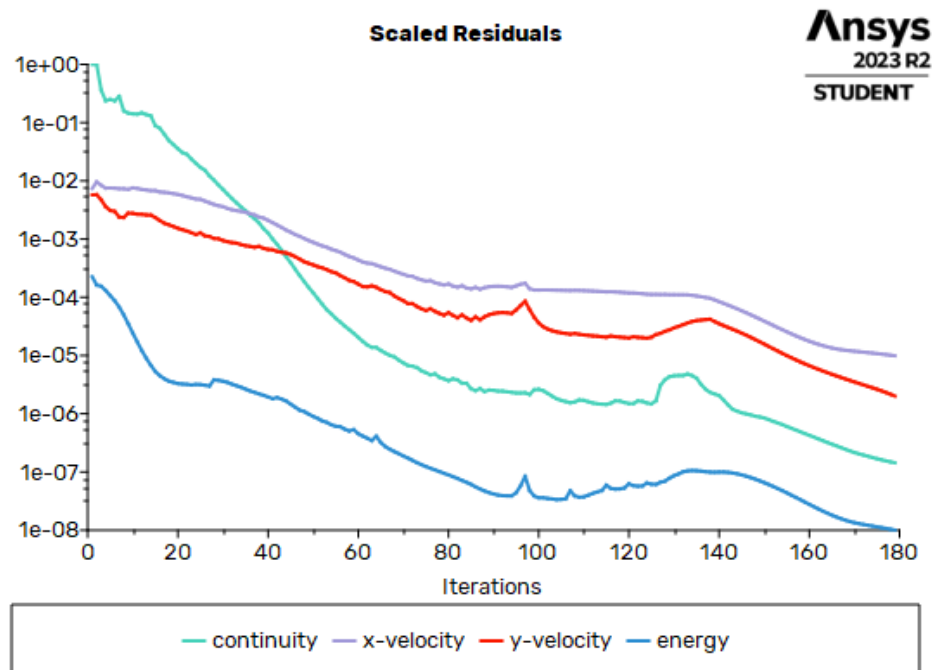


Geometric Model of the fin



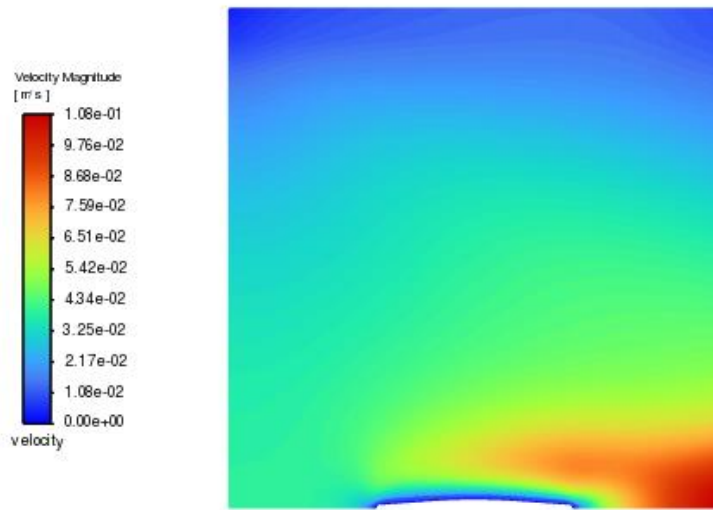
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Solvent Domain

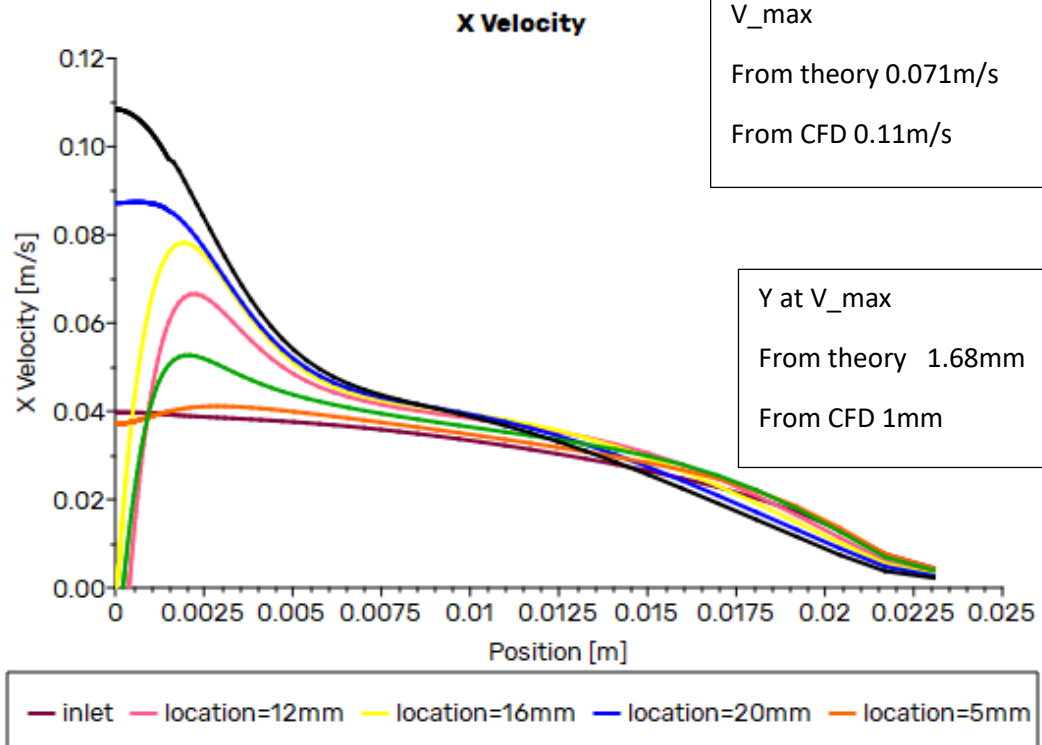


Static temperature contour



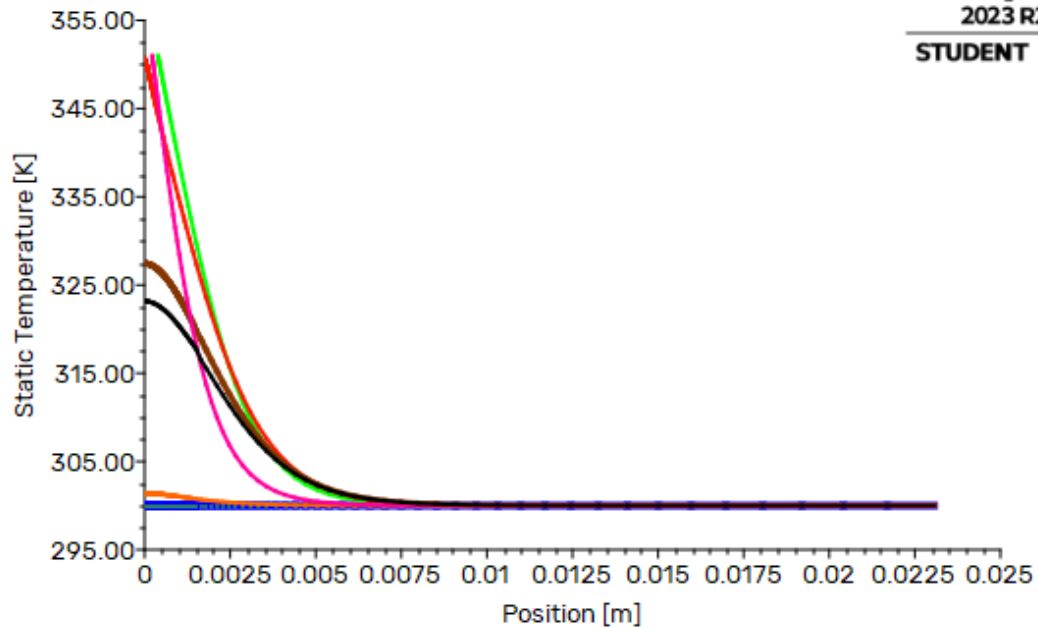


Velocity contour



### Static Temperature

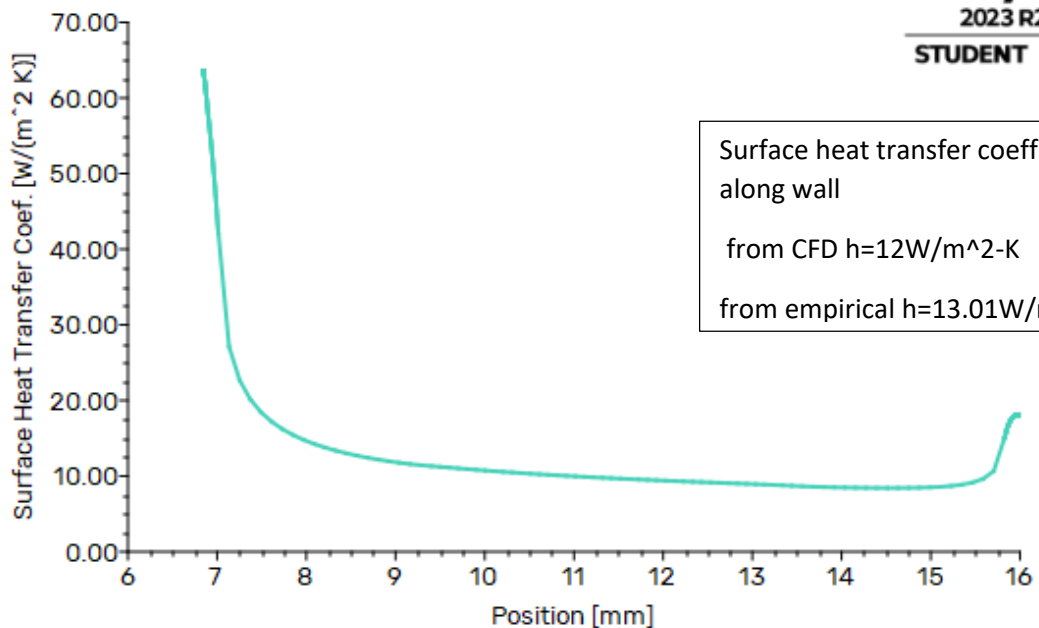
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■ inlet ■ location=12mm ■ location=16mm ■ location=20mm ■ location=5mm

### Surface Heat Transfer Coef.

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Surface heat transfer coefficient  
along wall

from CFD  $h=12\text{W/m}^2\text{-K}$

from empirical  $h=13.01\text{W/m}^2\text{-K}$

— wall

