

**ME 610 -- HW1B**

**Clamped-Clamped Beam using NASTRAN**

**Victor Cavalcanti**

## OBJECTIVE

The objective of this assignment is to construct a simple FEM model of a clamped-clamped beam using NASTRAN, and then to validate the results analytically. Two models will be made, one with 15 elements and one with 30 elements. The beam has dimensions of 40" x 0.25" x 1" (l x h x w). The beam is constructed of aluminum,  $E=10^7$  psi,  $\rho = 0.098$  lb/in<sup>3</sup>.

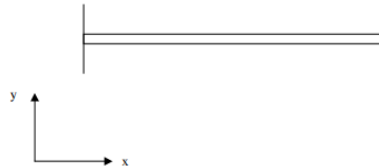


Figure 1- Model of the clamped-clamped beam.

## PROCEDURE

NASTRAN was used to build the model. The nodes were equally spaced for both the 15 and 30 element models. For the bulk deck, the CBAR elements were made to align with the global coordinate system pictured above. Examples of the GRID, CBAR, PBAR and MAT1 commands for the bulk deck are shown below:

```

$      ID      CP      X1      X2      X3      CD
GRID   101      0      0.0      0.0      0.0      0
GRID   102      0      2.667      0.0      0.0      0
GRID   103      0      5.333      0.0      0.0      0
GRID   104      0      8.000      0.0      0.0      0
$      EID      PID      GA      GB      X1      X2      X3
CBAR    101      100      101      102      0.0      1.0      0.0
$      ID      MAT      A      I1      I2      J
PBAR    100      100      0.25      1.30E-3      2.08E-2      4.38E-3
$      ID      E      G      NU      RHO
MAT1    100      10E6      .33      0.098

```

The model was constrained to motion in the x-y plane, with both its end nodes completely constrained. This was done by specifying the ASET and SPC1 commands in the '\*.dat' file:

```

$      C      GID1      "THRU"      GID2
ASET1   26      102      THRU      115
$      SID      C      GID1      GID2
SPC1    10      123456      101      116

```

The next command is used to allow for imperial units.

```
PARAM    WTMASS    .00259
```

The first 20 modes and frequencies were extracted from the models, along with the models corresponding mass and stiffness matrices. The modes and frequencies were calculated using MATLAB and the first 5 frequencies were compared.

The natural frequency was compared to the exact analytical solution, given by:

$$\omega_{n-analytical} = \frac{\lambda_i^2}{2\pi L} \sqrt{\frac{EI}{\bar{m}}} \quad (1)$$

Where  $\lambda_i = 4.73004074$ , and  $\bar{m} = A\rho$ , which is the mass per unit length. It was also compared to the natural frequency acquired by the Rayleigh quotient.

$$\omega_n^2 = \frac{U_{max}}{T_{max}} \quad (2)$$

$$U_{max} = \int_0^L \frac{1}{2} EI(x) \left( \frac{\delta^2 \phi(x)}{dx^2} \right)^2 dx \quad (3)$$

$$T_{max} = \int_0^L \frac{1}{2} \omega_n^2 \bar{m}(x) \phi^2 dx \quad (4)$$

Where we are using an assumed shape of  $\phi$ :

$$\phi = 1 - \cos\left(\frac{2\pi x}{L}\right) \quad (5)$$

Which satisfies all 4 boundary conditions ( $\phi(0) = 0, \phi(L) = 0, \phi'(0) = 0, \phi'(L) = 0$ ).

When equation (5) is plugged into equations (2-4), we get:

$$\omega_n^2 = \frac{EI}{3\bar{m}} \left( \frac{2\pi}{L} \right)^4 \quad (6)$$

**RESULTS**

Below are the tabulated results for the first 5 frequencies from the 15 and 30 element models, both for the NASTRAN and for the MATLAB calculations, along with the % difference between them. The %difference is relative to  $\omega_{n-NASTRAN}$ , as shown below (**all frequencies in Hz**):

$$\% \text{ difference} = \frac{\omega_n - \omega_{n-NASTRAN}}{\omega_{n-NASTRAN}} * 100\% \quad (7)$$

Frequency #	$\omega_{n-(15)MATLAB}$	$\omega_{n-(15)NASTRAN}$	% diff	$\omega_{n-(30)MATLAB}$	$\omega_{n-(30)NASTRAN}$	% diff
1	31.855	31.855	0.000	31.855	31.854	0.000
2	87.812	87.812	0.000	87.808	87.808	0.000
3	172.172	172.172	0.000	172.140	172.140	0.000
4	284.706	284.706	0.000	284.563	284.563	0.000
5	425.577	425.577	0.000	425.106	425.106	0.000

Table 1- Frequencies acquired via NASTRAN and MATLAB. No difference was found between them.

The table above shows that the eigensolver for NASTRAN and MATLAB yields the same results. The table below shows the % error when the frequencies are compared to the analytical solution (note the table has been shortened, since there is no difference between MATLAB and NASTRAN).

Frequency #	$\omega_n - \text{exact}$	$\omega_n - \text{Rayleigh}$	% error	$\omega_n - 15$	% error	$\omega_n - 30$	% error
1	31.87	32.47	1.875	31.85	-0.07	31.85	-0.069
2	87.86	-	-	87.81	-0.06	87.81	-0.064
3	172.25	-	-	172.2	-0.05	172.17	-0.05
4	185.60	-	-	284.7	53.39	284.71	53.39
5	425.36	-	-	425.6	0.049	425.57	0.049

Table 2- Frequency error between the analytical frequencies and those found via NASTRAN/MATLAB.

Finally, the first 5 modes were plotted. The x-axis has been remapped to the physical location of the DOF.

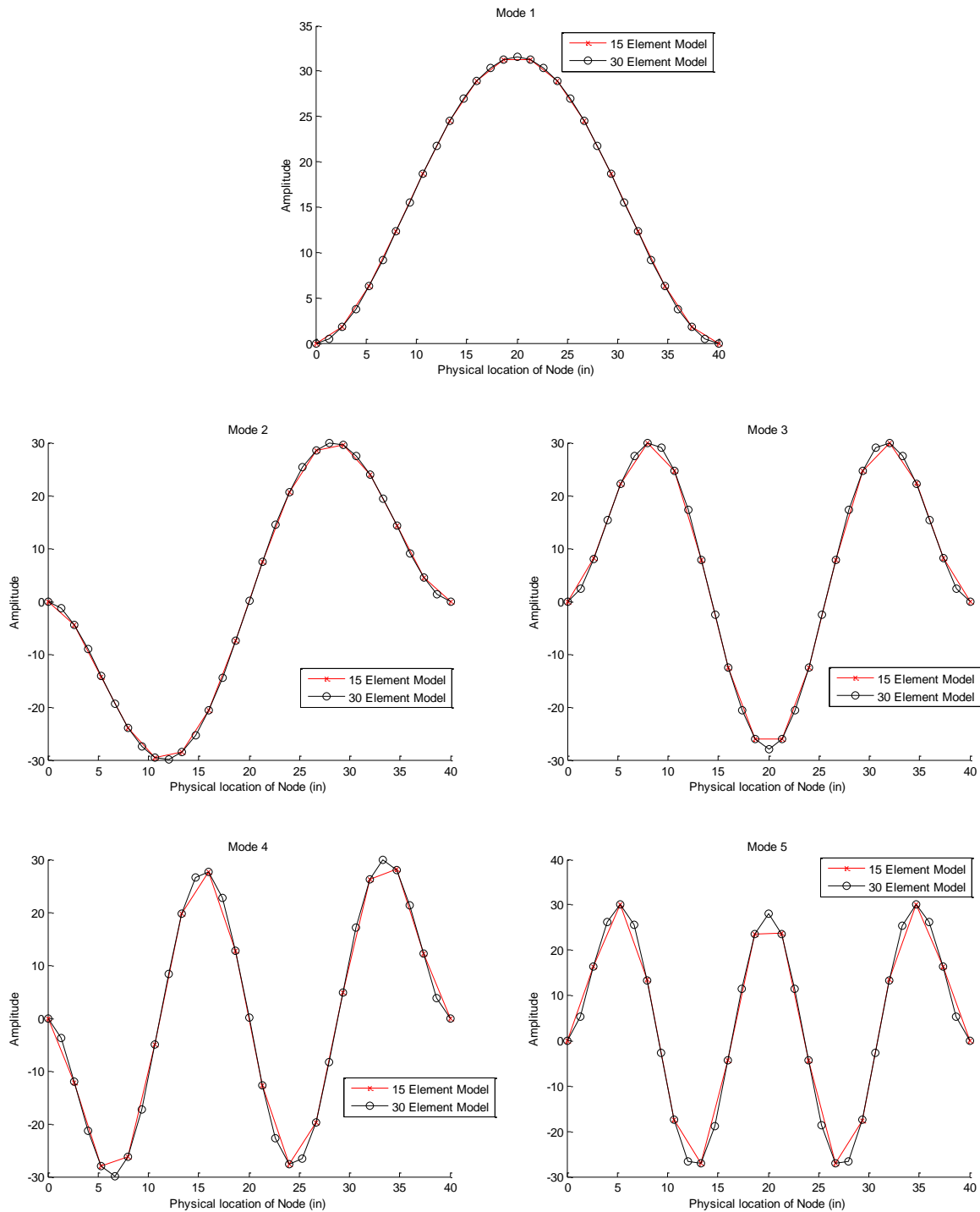


Figure 2- First 5 modes for both the 15 and 30 element beam.

## APPENDIX A – MATLAB & NASTRAN

### MATLAB

```

close all; clc; clear all;
load('hw1binfo');

% E = 10^7;
% I = 0.001302;
% p = 0.098;
% l = 40;
% Physical info in metric
A = 0.00016129;
E = 6.894*10^10;
I = 5.4193*10^-10;
p = 2712.63;
l=1.016;
w1 = sqrt(((E*I)/(3*A*p))*((2*pi)/l)^4/(2*pi));
lam = [ 4.73004074; 7.85320462; 10.9956079; 11.41371655; 17.2787597];
wans = (lam.^2./(2*pi*l^2)).*(E*I/(A*p))^0.5;

phys15 = 0:(2+(2/3)):40;
phys30 = 0:(1+(1/3)):40;
[phi15,v15] = eig(K15,M15);
[phi30,v30] = eig(K30,M30);
wnMAT15 = sqrt(diag(v15));
wnMAT30 = sqrt(diag(v30));
[wnMAT15,w15ind] = sort(wnMAT15,'ascend');
[wnMAT30,w30ind] = sort(wnMAT30,'ascend');
wn15hrz = wnMAT15/(2*pi);
wn30hrz = wnMAT30/(2*pi);

figure; hold on;
% subplot(1,2,1);hold on;
plot(phys15,[ 0; phi15(1:2:end,1); 0],'rx-');
plot(phys30,[ 0; phi30(1:2:end,1); 0],'ko-');
title('Mode 1');
ylabel('Amplitude');
xlabel('Physical location of Node (in)');
legend('15 Element Model', '30 Element Model');

figure; hold on;

% subplot(1,2,2); hold on;
plot(phys15,[ 0; phi15(1:2:end,2); 0],'rx-');
plot(phys30,[ 0; phi30(1:2:end,2); 0],'ko-');
title('Mode 2');
ylabel('Amplitude');
xlabel('Physical location of Node (in)');
legend('15 Element Model', '30 Element Model');

figure; hold on;
% subplot(1,3,1); hold on;
plot(phys15,[ 0; phi15(1:2:end,3); 0],'rx-');
plot(phys30,[ 0; phi30(1:2:end,3); 0],'ko-');
title('Mode 3');
ylabel('Amplitude');
xlabel('Physical location of Node (in)');
legend('15 Element Model', '30 Element Model');

figure; hold on;

% subplot(1,3,2); hold on;
plot(phys15,[ 0; phi15(1:2:end,4); 0],'rx-');
plot(phys30,[ 0; phi30(1:2:end,4); 0],'ko-');
title('Mode 4');
ylabel('Amplitude');
xlabel('Physical location of Node (in)');
legend('15 Element Model', '30 Element Model');

figure; hold on;

% subplot(1,3,3); hold on;
plot(phys15,[ 0; phi15(1:2:end,5); 0],'rx-');
plot(phys30,[ 0; phi30(1:2:end,5); 0],'ko-');
title('Mode 5');
ylabel('Amplitude');
xlabel('Physical location of Node (in)');
legend('15 Element Model', '30 Element Model');

```

**Beam15.blk**

```

$
ASSIGN MASTER='beam15.MASTER'
ASSIGN DBALL ='beam15.DBALL'
$
ID   BEAM,MODES
SOL  103  $ Normal modes - superelements
TIME 30   $ 30 CPU minutes
DIAG 8    $ Print matrix trailers
DIAG 20   $ Print database fetch/store messages
$
$
CEND
TITLE   =FIXED-FIXED 15 ELE BEAM
SUBTITLE =FINITE ELEMENT MODEL (FEM)
LABEL   =NORMAL MODES ANALYSIS
$
ECHO = UNSORT      $ Print unsorted bulk data deck
$
SEALL = ALL        $ Required for SOL 103
SUPER = ALL        $ Required for SOL 103
$
SPC = 10           $ Constrain to plane motion
METHOD = 70        $ First 20 modes
$
DISP(PLOT) = ALL   $ Recover but do not print mode shapes
$
BEGIN BULK
$
$
$ PARAMETER CARDS
$ -----
$
PARAM AUTOSPC YES
PARAM GRDPNT 0
PARAM USETPRT 0
PARAM WTMASS .00259
PARAM COUPMASS1
PARAM POST 0
$
$
$ COMPUTE EIGENVALUES USING THE LANCZOS METHOD
$ -----
$
$   SID  F1  F2  ND
EIGRL 70          20
$
$ SPECIFY ASET DOF
$ -----
$   C   GID1  "THRU" GID2
ASET1 26  102  THRU  115
$
$ CONSTRAIN OUT-OF-PLANE MOTION
$ -----
$   SID  C   GID1  "THRU" GID2
$   SID  C   GID1  GID2
$   SID  C   GID1  GID2
SPC1 10  123456 101  116
$
$
$ BEAM BULK DATA
$ -----
$
INCLUDE 'beam15.blk'
$
ENDDATA

```

**Beam15.dat**

```

$ GRID POINTS
$ -----
$
$ 2 3 4 5 6 7 8 9
$
$ ID CP X1 X2 X3 CD
GRID 101 0 0.0 0.0 0.0 0
GRID 102 0 2.667 0.0 0.0 0
GRID 103 0 5.333 0.0 0.0 0
GRID 104 0 8.000 0.0 0.0 0
GRID 105 0 10.66 0.0 0.0 0
GRID 106 0 13.33 0.0 0.0 0
GRID 107 0 16.00 0.0 0.0 0
GRID 108 0 18.66 0.0 0.0 0
GRID 109 0 21.33 0.0 0.0 0
GRID 110 0 24.00 0.0 0.0 0
GRID 111 0 26.66 0.0 0.0 0
GRID 112 0 29.33 0.0 0.0 0
GRID 113 0 32.00 0.0 0.0 0
GRID 114 0 34.66 0.0 0.0 0
GRID 115 0 37.33 0.0 0.0 0
GRID 116 0 40.00 0.0 0.0 0
$GRID 117 0 2.786 0.0 0.0 0
$GRID 118 0 2.960 0.0 0.0 0
$GRID 119 0 3.135 0.0 0.0 0
$GRID 120 0 3.309 0.0 0.0 0
$GRID 121 0 3.483 0.0 0.0 0
$GRID 122 0 3.657 0.0 0.0 0
$
$ BAR ELEMENTS
$ -----
$ 2 3 4 5 6 7 8 9
$
$ EID PID GA GB X1 X2 X3
CBAR 101 100 101 102 0.0 1.0 0.0
CBAR 102 100 102 103 0.0 1.0 0.0
CBAR 103 100 103 104 0.0 1.0 0.0
CBAR 104 100 104 105 0.0 1.0 0.0
CBAR 105 100 105 106 0.0 1.0 0.0
CBAR 106 100 106 107 0.0 1.0 0.0
CBAR 107 100 107 108 0.0 1.0 0.0
CBAR 108 100 108 109 0.0 1.0 0.0
CBAR 109 100 109 110 0.0 1.0 0.0
CBAR 110 100 110 111 0.0 1.0 0.0
CBAR 111 100 111 112 0.0 1.0 0.0
CBAR 112 100 112 113 0.0 1.0 0.0
CBAR 113 100 113 114 0.0 1.0 0.0
CBAR 114 100 114 115 0.0 1.0 0.0
CBAR 115 100 115 116 0.0 1.0 0.0
$CBAR 116 100 116 117 0.0 1.0 0.0
$CBAR 117 100 117 118 0.0 1.0 0.0
$CBAR 118 100 118 119 0.0 1.0 0.0
$CBAR 119 100 119 120 0.0 1.0 0.0
$CBAR 120 100 120 121 0.0 1.0 0.0
$CBAR 121 100 121 122 0.0 1.0 0.0
$
$ BAR PROPERTIES
$ -----
$ 2 3 4 5 6 7 8 9
$ ID MAT A I1 I2 J
PBAR 100 100 0.25 1.30E-3 2.08E-2 4.38E-3
$ ID E G NU RHO
MAT1 100 10E6 .33 0.098

```