


THAPAR INSTITUTE
 OF ENGINEERING & TECHNOLOGY
 (Deemed to be University)

Mass Transfer-I

Mass Transfer Analogies



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Analogies among Mass, Heat, and Momentum Transfer

Analogies

Heat \Leftrightarrow Mass \Leftrightarrow (sometimes) Momentum

Analogies are useful tools

1. An aid to understand transfer phenomena
2. A sound means to predict behavior of systems for which limited quantitative data are available



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Molecular Transport Equations

Rate of transport = $\frac{\text{Driving force}}{\text{Resistance}}$

$$\tau_{yx} = -\nu \frac{d(v_x \rho)}{dy}$$

MOMENTUM
Newton's law

$$\frac{q_y}{A} = -\alpha \frac{d(\rho c_p T)}{dy}$$

HEAT
Fourier's law

$$J_{Ay}^* = -D_{AB} \frac{dc_A}{dy}$$

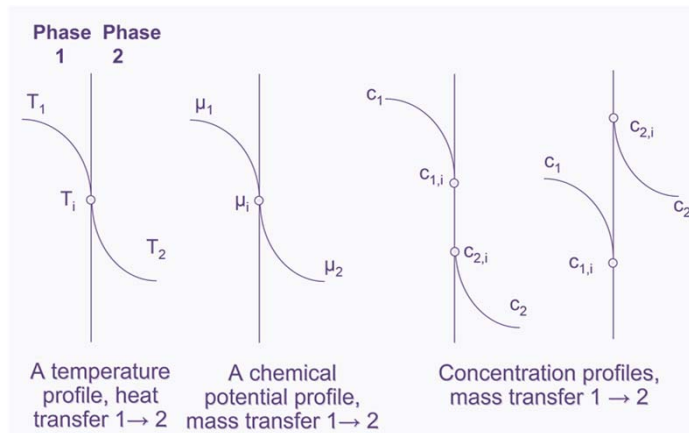
MASS
Fick's law



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Analogous quantities in transport phenomena

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	Momentum	Mass	Energy
Transport quantity per volume	ρu_x	C_A	$\rho C_p T$
Transport coefficient	μ [$\text{g}\cdot\text{cm}^{-1}\text{s}^{-1}$]	D_{AB} [$\text{cm}^2\cdot\text{s}^{-1}$]	k [$\text{cal}\cdot\text{cm}^{-1}\text{s}^{-1}\text{K}^{-1}$]
Diffusivity [$\text{cm}^2\cdot\text{s}^{-1}$]	$\nu = \mu/\rho$	D_{AB}	$\alpha = k/\rho C_p$
Flux law	$\tau_{xz} = -\nu \frac{d}{dz}(\rho u_x)$	$J_z = -D_{AB} \frac{d}{dz} C_A$	$q_z = -\alpha \frac{d}{dz}(\rho C_p T)$
Dimensionless transport groups	$Re = \frac{UL}{\nu}$	$Sc = \frac{\nu}{D_{AB}}$	$Pr = \frac{\nu}{\alpha} \left(= \frac{C_p \mu}{k} \right)$

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Reynolds Analogy

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Assumptions

- Only turbulent core is present
- Velocity, temperature and concentration profiles are perfectly matching.
- All diffusivities (turbulent diffusivities) are same
- The molecular diffusivities are negligible.

$$(\alpha) = (D_{AB}) = \left(\frac{\mu}{\rho} \right)$$

When all the three diffusivities are equal, then

Prandtl Number (N_{Pr}) = Schmidt number (N_{Sc}) = 1.

Reynolds analogy equation is

$$\left(\frac{f}{2} \right) = \frac{h}{\rho C_p u_0} = \frac{k_c}{u_0}$$

Where,

f = Friction factor
 h = Heat transfer coefficient
 α = Thermal diffusivity
 C_p = Heat capacity
 D_{AB} = Mass diffusivity
 k_c = Mass transfer coefficient
 u_0 = Characteristic velocity
 ρ = Density
 St = Stanton number
 Pr = Prandtl number
 Sc = Schmidt number
 Re = Reynolds number
 Sh = Sherwood number

$$St = \frac{Sh}{Re \cdot Sc} = \frac{f}{2}$$

$$St = \frac{Nu}{Re \cdot Pr} = \frac{f}{2}$$

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The Reynolds analogy is not applicable for following cases

- For other fluids, where $Pr \neq Sc \neq 1$; usually the case for liquids
- We **CANNOT** neglect molecular diffusivities in case of boundary layer, where diffusion, conduction, and viscosity are important

Experimental results show that the above equation

- Correlate data approximately for gases in turbulent flow for $0.6 < Pr$ (for gases) < 2.5
- DOES NOT correlate experimental data for liquids in turbulent flow
- DOES NOT correlate experimental data for any fluids in laminar flow

CONCLUSIONS

1. At $Pr = Sc = 1$, the mechanisms for mass, heat, and momentum are identical
2. For other fluids, transfer processes differ in some manner functionally related to the Pr and Sc numbers.
3. The Reynolds analogy is valid ONLY at $Pr = Sc = 1$

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Chilton-Colburn Analogy

Assumptions

- Only turbulent core is present
- Velocity, temperature and concentration profiles are perfectly matching.
- $N_{Pr} \neq N_{Sc} \neq 1$

Analogy equation is

$$\left(\frac{f}{2}\right) = \left(\frac{k_c}{u_0}\right)(Sc)^{2/3} = \frac{h}{(\rho C_p u_0)}(Pr)^{2/3}$$

$$St \approx \left(\frac{f}{2}\right) \cdot (Sc)^{-2/3}$$

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Taylor-Prandtl Analogy

Assumptions

- turbulent and laminar sublayer are present
- $N_{Pr} \neq N_{Sc} \neq 1$

Analogy equation is

$$\frac{k_c}{u_0} = \frac{h}{\rho C_p u_0} = \frac{\left(\frac{f}{2}\right)}{\left[1 + 5\sqrt{\frac{f}{2}}(Sc - 1)\right]} = \frac{\left(\frac{f}{2}\right)}{\left[1 + 5\sqrt{\frac{f}{2}}(Pr - 1)\right]}$$

$$St \cong \frac{\frac{1}{2}f}{1 + 5\left(\frac{1}{2}f\right)^{1/2}(Pr - 1)}$$

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Von-Karman Analogy

Assumptions

- Turbulent, laminar sublayer and buffer layers are present
- Universal velocity profile equations are applicable
- $N_{Pr} \neq N_{Sc} \neq 1$

Analogy equation is

$$\frac{k_c}{u_0} = \frac{\frac{f}{2}}{1 + 5\sqrt{\frac{f}{2}} \left[(Sc - 1) + \ln \left(\frac{5Sc + 1}{6} \right) \right]}$$

$$= \frac{\frac{f}{2}}{1 + 5\sqrt{\frac{f}{2}} \left[(Pr - 1) + \ln \left(\frac{5Pr + 1}{6} \right) \right]}$$

$$St_m \approx 0.08 \cdot \left(\frac{c_f}{2}\right)^{1/2} \cdot (Sc)^{-0.704}$$

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Example

A 1 m² thin plate of solid naphthalene is oriented parallel to a stream of air flowing at 30 cm/s. The air is at 300 K and 1 atm pressure. The plate is also at 300 K. Determine the rate of sublimation from the plate. The diffusivity of naphthalene in air at 300 K and 1 atm is 5.9×10^{-4} m²/s. Vapour pressure of naphthalene at 300 K is 0.2 mm Hg.

Solution

$$u_0 = 30 \text{ cm/s}, T = 300 \text{ K}, P_t = 1 \text{ atm}, D_{AB} = 5.9 \times 10^{-4} \text{ m}^2/\text{s},$$

$$p_A = 0.2 \text{ mm Hg}$$

$$\rho_{\text{air}} = 1.15 \times 10^{-3} \text{ g/cc}, \mu_{\text{air}} = 0.0185 \text{ cp}, D = 1 \text{ m (Length)}$$

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$$N_{Sc} = \frac{\mu}{\rho D_{AB}} = \frac{0.0185 \times 10^{-3}}{1.15 \times 10^{-3} \times 10^3 \times 5.9 \times 10^{-4}} = 0.0273 \neq 1$$

So we will use Chilton analogy,

$$N_{Re} = \frac{D \bar{V} \rho}{\mu} = \frac{1.15 \times 10^{-3} \times 10^3 \times 1 \times 0.3}{0.0185 \times 10^{-3}} = 18648.65$$

So flow is turbulent, so Chilton-Colburn analogy can be used.

$$\frac{f}{2} = \frac{k_c}{u_0} (N_{Sc})^{2/3}$$

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$$f = 0.072 \times (N_{Re})^{-0.25}$$

$$f = 6.161 \times 10^{-3}$$

$$k_c = \frac{f \times u_0}{2(N_{Sc})^{2/3}}$$

$$k_c = \frac{6.161 \times 10^{-3} \times 0.3}{2(0.0273)^{2/3}} = 0.0102 \text{ m/s}$$

$$N_A = k_c (C_{A1} - C_{A2}) = k_c \frac{(p_{A1} - p_{A2})}{RT}$$

$$= \frac{0.0102 \times 1.0133 \times 10^5}{8314 \times 300} \left[\left(\frac{0.2}{760} \right) - 0 \right]$$

$$N_A = 1.09 \times 10^{-7} \text{ kmol/m}^2 \text{ s.} \quad \text{Ans.}$$

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References

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ETH

Eidgenössische Technische Hochschule Zürich

Institute for Process Engineering

Mass Transfer

Theories for Mass Transfer Coefficients

Lecture 9, 15.11.2017, Dr. K. Wegner

- Lecture notes/ppt of Dr. Yahya Banat (ybanat@qu.edu.qa)

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