Hydrodynamic and chemical effects of hydrogen dilution on soot evolution in turbulent nonpremixed bluff body ethylene flames

Sili Deng^{a,*}, Michael E. Mueller^a, Qing N. Chan^b, Nader H. Qamar^c, Bassam B. Dally^d, Zeyad T. Alwahabi^d, Graham J. Nathan^d

^aDepartment of Mechanical and Aerospace Engineering, Princeton University, Princeton, USA
^bSchool of Mechanical and Manufacturing Engineering, The University of New South Wales, Australia
^cFCT-Combustion, Australia
^dCentre for Energy Technology, The University of Adelaide, Australia

Abstract

A turbulent nonpremixed bluff body ethylene/hydrogen (volume ratio 2:1) flame was investigated using a combination of experiments and Large Eddy Simulations, and compared with a neat ethylene counterpart [Mueller *et al.* Combust. Flame, 160, 2013]. Similar to the ethylene buff body flame, a low-strain recirculation zone, a high-strain neck region, and a downstream jet-like region were observed. However, the maximum soot volume fraction in the recirculation zone and jet-like region of the hydrogen diluted case are significantly lower than the ethylene case. Flamelet calculations demonstrated that hydrogen dilution suppressed soot formation due to the reduction of C/H ratio, and a six times soot volume fraction decrease was estimated based on this chemical effect. The downstream jet-like region shows a reduction in soot consistent with this analysis. However, soot reduction in the recirculation zone is substantially larger than this analysis suggested, indicating an additional hydrodynamic effect. Large Eddy Simulation was used to further investigate soot evolution in the recirculation zone and to elucidate the role of hydrogen dilution. Since the central jet Reynolds numbers in both cases are the same (approximately 30,900), the jet velocity of the hydrogen diluted case is higher, resulting in a shorter and leaner recirculation zone, which inhibits soot formation and explains the larger magnitude in soot reduction.

Keywords: Soot, Laser-induced incandescence, Large Eddy Simulation (LES), Bluff body flame, Turbulent nonpremixed flame

^{*}Corresponding Author: silideng@princeton.edu

1. Introduction

Due to the importance of soot in practical transportation, propulsion, and power generation systems, the formation and growth of soot have been extensively studied. Most of these studies focus on laminar configurations since flow conditions are better controlled and characterized, which enables detailed analysis of soot dynamics [1]. Nonetheless, practical devices operate at turbulent conditions. Soot evolution in turbulent reacting flows and the small-scale interactions among soot, turbulence, and chemistry has been aided by Direct Numerical Simulation (DNS). However, these studies are limited to two-dimensional, often with reduced chemistry and empirical soot model [2–5], due to the inherent complexity of chemical mechanisms, and hence, computational cost. Recently, Attili *et al.* [6] performed the first three-dimensional DNS of turbulent nonpremixed jet flames employing a high-order statistical model of soot and detailed chemical mechanism, which includes the soot precursor naphthalene, and investigated Damköhler number effects on soot formation and growth [7]. Nevertheless, the Reynolds number is limited to 15, 000.

To investigate jet flames at high Reynolds numbers, a combination of experiments and Large Eddy Simulation (LES) has been used [8–10], however, the jet flame configuration does not contain more complex fluid dynamics found in practical combustion systems such as recirculating flow. To bridge this gap, recent experiment and LES has been used for recirculating flows in canonical geometries. Mueller *et al.* [11] experimentally and computationally studied a turbulent nonpremixed bluff body ethylene flame. The LES model for sooting flames was validated for a piloted natural gas jet diffusion flame, the Delft III flame [8], and ethylene was chosen in the bluff body study to avoid large uncertainties in soot precursor chemistry in methane flames. Laser-induced incandescence (LII) was used to measure soot volume fraction.

In the present work, a hydrogen diluted ethylene sooting flame with the same bluff body configuration is investigated. During the thermal decomposition of hydrocarbon fuels, it has been shown that the dilution of hydrogen slows down the formation of soot [12]. Extensive laminar studies have been conducted with simplified flow conditions to understand the overall suppression of soot formation in hydrogen diluted diffusion flames and have attributed such suppression to both dilution and chemistry effects [13–17]. In addition, to maintain the same Reynolds number as the ethylene bluff body flame [11], a faster central jet is needed, which will change the fuel to coflow air momentum flux ratio and affect hydrodynamics.

The objective of the this investigation is threefold: first, to understand the evolution of soot in the hydrogen diluted ethylene bluff body flame utilizing a combination of experiments and computations; second, to assess differences between hydrogen diluted and neat ethylene flames and further validate the LES model; and, third, to differentiate the hydrodynamic and chemical effects by hydrogen addition.

2. Experimental methodology

The experimental setup used in the current study is similar to previous bluff body studies [18, 19] and was kept the same as the previous ethylene case [11]. In brief, the outer diameter of the bluff body (D_B) is 50 mm, and the

diameter (D_J) of the central fuel jet is 3.6 mm, from which an ethylene/hydrogen mixture (2:1 by volume) is issued at 102.1 m/s. The central jet Reynolds number was kept the same as the previous study [11] at 30,900. The bluff body burner was mounted in a contraction with an exit cross section of 150 by 150 mm², from which air coflow is issued at 23 m/s.

The 1064 nm beam from an Nd:YAG laser was used for LII excitation. The laser sheet has a height of 80 mm through the measurement volume and has a thickness of 0.3 mm. The operating LII fluence was kept at 0.9 J/cm² to ensure the independence of the laser fluence variation [20, 21]. A Gaussian distribution of the spatial fluence with a 8% standard deviation was achieved. All images presented in this work have been clipped at the edges where the laser sheet was found to exhibit low fluence.

The LII signal was filtered at 430 ± 10 nm and detected by an intensified CCD camera. A short gate width of 40 ns was used to reduce the size-dependent sensitivity of the signal [22]. The LII signal has been calibrated by laser extinction measurements previously [11]. With this system, the in-plane resolution of the images is 0.26 mm/pixel in each direction, and the detection threshold is about 3 ppb.

The data presented in this work have been corrected for background interference and detector attenuation. According to the previous ethylene bluff body study, the estimated measurement uncertainty is about 25%.

3. Computational methodology

The modeling of soot-chemistry-turbulence interactions is aided by a statistical soot model, a modified Radiation Flamelet/Progress Variable (RFPV) combustion model for sooting flames, and a presumed subfilter PDF for closure. Complete details of the integrated LES model for sooting turbulent nonpremixed flames can be found in Mueller and Pitsch [8] and the references therein.

In brief, soot particles and aggregates are described by their volume (V) and surface area (S), and transport equations are solved for the bivariate moment, $M_{x,y} = \sum_i V_i^x S_i^y N_i$, where x and y are the moment orders for volume and surface area and summation over i implies summation over all particle sizes. Comprehensive physical and chemical processes governing the evolution of the moments are considered: particle nucleation from Polycyclic Aromatic Hydrocarbon (PAH) dimers [23–25], PAH condensation, particle coagulation [26], surface growth by C_2H_2 -addition (HACA) mechanism [27], oxidation [28, 29], and oxidation-induced fragmentation [30]. Moment closure is achieved with the Hybrid Method of Moments (HMOM) [26]. In total, four quantities are used to describe the evolution of the soot population: the total soot number density ($M_{0,0}$), the total soot volume ($M_{1,0}$), the total soot surface area ($M_{0,1}$), and the number density of smaller particles (N_0).

The thermochemical states, such as temperature, species mass fractions (Y), and other derived quantities, are obtained from tabulated chemistry, described with the RFPV model of Ihme and Pitsch [31] with modifications for soot by Mueller and Pitsch [8]. Solutions of the steady flamelets equations are parameterized by the mixture fraction (Z), a reaction progress variable $(C = Y_{CO_2} + Y_{CO} + Y_{H_2O} + Y_{H_2O} + Y_{H_2O})$, and a heat loss parameter (H), to account for heat losses

due to radiation. Due to significant unsteady effects for PAH [5], the mass fractions for these species deviate from their steady values in the flamelet database. Therefore, an additional transport equation for a lumped PAH species is solved.

The closure for filtered quantities such as density, gas-phase source terms, and soot source terms are achieved with a presumed subfilter PDF model by Mueller and Pitsch [8, 32]. The joint subfilter PDF of the mixture fraction, progress variable, heat loss parameter, and soot moments ($\tilde{P}(Z,C,H,M_i)$) can be modeled by the product of the thermochemical subfilter PDF, $\tilde{P}(Z,C,H)$, and the soot subfilter PDF, $\tilde{P}(M_i)$, due to the time scale decoupling for the gas-phase and soot evolution [32]. The thermochemical PDF is modeled with a beta distribution for the mixture fraction [33]. Convolution of the flamelet database with the PDF is done *a priori* and tabulated as a function of the filtered mixture fraction, subfilter mixture fraction variance, filtered progress variable, and filtered heat loss parameter. The subfilter mixture fraction variance is obtained from the solution of a transport equation for the filtered squared of the mixture fraction with a linear relaxation model [34] for the subfilter scalar dissipation rate. The soot subfilter PDF is modeled with a double delta distribution [32], which requires solving an additional transport equation for the filtered square of the number density.

In summary, the continuity, momentum equations, and transport equations for \widetilde{Z} , \widetilde{Z}^2 , \widetilde{C} , \widetilde{H} , \widetilde{Y}_{PAH} , $\overline{M}_{0,0}$, $\overline{M}_{1,0}$, $\overline{M}_{0,1}$, \overline{N}_0 , and $\overline{M}_{0,0}^2$ are solved in the simulation. All of the subfilter stresses and scalar fluxes are closed with dynamic models [35] with Lagrangian averaging [36]. The soot unknown moments are closed with HMOM, and density, gas-phase, and soot source terms are closed with presumed subfilter PDF.

The simulation details are similar to the previous ethylene bluff body flame [11]. Flamelet solutions were computed using FlameMaster [37] with the chemical mechanism, including PAH, of Pitsch and co-workers [38, 39]. The soot and combustion models were implemented in NGA [40]. The continuity and momentum equations are discretized with a centered, second-order scheme, and the scalar equations are discretized with a bounded QUICK scheme [41]. The computational domain is discretized on a structured, non-uniform grid, with $384 \times 192 \times 64$ points in the axial, radial, and circumferential directions, respectively. To be consistent with the neat ethylene case, turbulence intensity in the central jet is increased by 10%, and a turbulent boundary layer condition is specified for the coflow.

4. Results and discussion

4.1. Overall flame structure

The overall structure of the turbulent nonpremixed hydrogen diluted ethylene bluff body flame is shown and compared with the neat ethylene counterpart in Fig. 1. Qualitatively, three distinct regions are identified: a sooting recirculation zone, a non-sooting, high strain neck region, and a sooting jet-like region. Quantitatively, the clipped, time-averaged LII images of soot volume fraction are also compared. Soot is formed close to the bluff body, where the residence time is long and turbulence intensity is low. No detectable soot is convected to nor formed in the high strain neck region. Further downstream in the jet-like region, where the scalar dissipation rate decreases as mixing

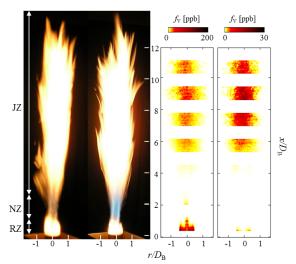


Figure 1: Photographs of the neat ethylene (left) and ethylene/hydrogen (right) bluff body flame, and the corresponding collages of the time-averaged LII images of the soot volume fraction distribution. Three distinct regions are indicated: RZ (recirculation zone), NZ (neck zone), and JZ (jet-like zone).

proceeds, soot formation once again occurs. Since the Reynolds numbers are the same in both cases, the overall flame lengths are similar, while the recirculation zone of the hydrogen diluted flame seems shorter.

Although the two flames share similar flow and soot formation patterns, the ethylene/hydrogen flame is significantly less sooting than the neat ethylene flame. As demonstrated in Fig. 2, the axial profiles of the radially integrated soot volume fraction (ppm-mm²) are compared for the two flames. Both profiles are bimodal, corresponding to the recirculation zone and jet-like region, respectively. However, both peaks in the hydrogen diluted flame is reduced. Furthermore, the reduction in the integrated soot volume fraction is more pronounced in the recirculation zone, compared to the downstream jet-like region. Specifically, according to both Figs. 1 and 2, the soot reduction in the recirculation zone is more than an order of magnitude, while, in the jet-like zone, the reduction is about a factor of six. The difference in the soot reduction rate with hydrogen dilution indicates different soot reduction mechanisms and different roles that hydrogen dilution plays in these two regions.

4.2. Effects of hydrogen dilution

Two effects of hydrogen dilution are potentially relevant: chemical and hydrodynamic effects. Chemically, due to the reduced C/H ratio, PAH formation will be inhibited in the hydrogen diluted flame, and therefore, soot formation is inhibited. Hydrodynamically, since the fuel jet velocity is increased to match the Reynolds number, the fuel to air coflow momentum flux ratio is also increased. As Dally *et al.* [42] demonstrated, the increase in the fuel jet momentum flux decreases the strength of the mixture in the outer vortex in the recirculation zone and shifts the stoichiometric mixture fraction contour closer to central jet. Such a shift will result in a relatively leaner recirculation zone, and therefore further inhibit soot formation.

To justify and differentiate these effects, steady flamelets at a moderate scalar dissipation rate ($\chi_{st} = 10 \text{ s}^{-1}$) were calculated and the PAH mass fraction is shown in Fig. 3. PAH forms at rich mixture fractions (the stoichiometric

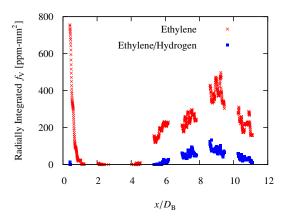


Figure 2: Total soot volume per unit height obtained from the radial integration of the time-averaged soot volume fraction at each height. For clarity, only every second measurement point is shown.

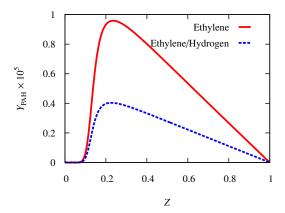
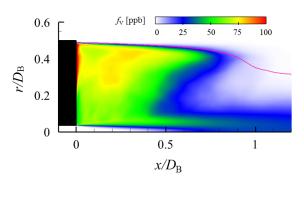


Figure 3: PAH mass fraction profile from the steady flamelet calculation at $\chi = 10 \text{ s}^{-1}$.

mixture fraction for both cases is about 0.06) and peaks at Z = 0.23. Y_{PAH} decreases at even richer mixture fractions due to the temperature drop. Comparing the peak values of Y_{PAH} , hydrogen dilution results in a decrease of a factor of 2.5. Previous DNS [5–7] and LES [8, 11] studies demonstrated that PAH based growth (PAH dimer nucleation and condensation) accounted for most of the soot mass while growth due to acetylene via the HACA mechanism was less important for the jet-like configuration. In addition, in the soot model adopted in this LES study, the PAH based growth rate scales with square of PAH concentration [8]. Consequently, roughly a factor of six decrease in soot volume fraction is expected due to the chemical effect. This agrees with trend in the reduction rate in the jet-like region, and, therefore, chemical effects are dominant in this region.

Conversely, noting that the chemical effect predicted by the steady flamelet solution cannot explain the additional soot reduction in the recirculation zone, soot evolution in this region is further analyzed with LES. The time-averaged soot volume fraction in the recirculation zone from LES is compared between the neat and hydrogen diluted ethylene flames in Fig. 4. For both cases, soot is confined to fuel-rich regions. In accordance with the trend of the radially integrated soot volume fraction in Fig. 2, significant soot reduction is observed in the hydrogen diluted case. Specif-



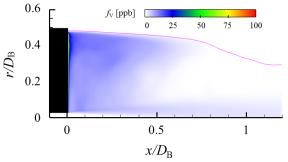
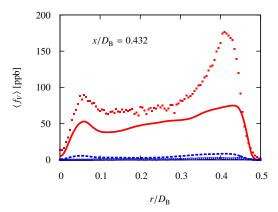


Figure 4: Time-averaged soot volume fraction [ppb] in the recirculation zone of the neat ethylene (top) and ethylene/hydrogen (bottom) flames. The magenta line correspond to the stoichiometric mixture fraction contour.

ically, radial profiles of the time-averaged soot volume fraction are compared for both cases at two axial locations in the recirculation zone, shown in Fig. 5.

Qualitatively, a bimodal radial distribution is found. Mueller $et\,al.$ [11] found in the ethylene bluff body flame that the inner and outer peak corresponds to the PAH-based growth and acetylene-based surface growth pathway, respectively. Quantitatively, the soot volume fraction of the ethylene case is slightly underpredicted, but within experimental uncertainty [11]. However, for the hydrogen diluted case, soot volume fraction is overpredicted. In the experiments, the most upstream laser measurement location is slightly less than $0.5D_{\rm B}$ due to the laser scattering from the bluff body surface. Moreover, due to the increased fuel jet velocity, the length of the recirculation zone is shorter. As a consequence, the measurement locations in the relatively high soot loading region within the recirculation zone are limited. Noting that in Fig. 2, the slope of the integrated soot volume fraction profile in the recirculation zone is very steep. Therefore, the experimental measurement in this region is particularly sensitive to the spatial resolution. Since the absolute soot volume fraction is low, the 3 ppb detection threshold might also contribute to the overprediction.

To elucidate the hydrodynamic effects on soot reduction, the mixture fraction radial profiles from LES as well as the characteristic time scales of the soot formation and oxidation processes from a slighted strained ($\chi_{st} = 0.01 \text{ s}^{-1}$) steady flamelet solution are shown in Fig. 6. The mixture fraction profile near the bluff body surface is affected substantially by hydrogen dilution, while the characteristic time scales of PAH-based growth, acetylene-based growth,



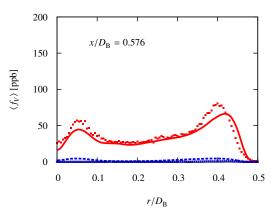
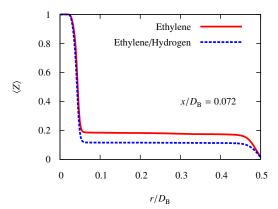


Figure 5: Radial profiles of the time-averaged soot volume fraction. Lines are the LES, and symbols are the measurements. The ethylene flame (red solid line and red closed symbol) is reproduced from [11], and the ethylene/hydrogen flame is from this work (blue dashed line and blue open symbol).



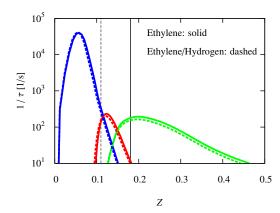


Figure 6: Left: radial profile of the time-averaged mixture fraction. Left: characteristic inverse time scales of the soot processes from the steady flamelet calculations at $\chi_{st} = 0.01 \text{ s}^{-1}$. Solid lines are for the ethylene flame, and dashed lines are for the ethylene/hydrogen flame. Oxidation process is in blue, surface growth is in red, and nucleation and condensation combined is in green. Black vertical lines correspond to the mixture fraction in the flat regions of the radial plot.

and oxidation processes remain the same. As shown in Fig. 6, in between the central jet and coflow air, the mixture fraction drops by 20%. Consequently, the mixture fraction shifts from where PAH-based growth is favored ($Z \sim 0.18$) to where oxidation becomes as fast as acetylene-based surface growth ($Z \sim 0.11$), suppressing soot formation and growth. Therefore, soot formation in the recirculation zone is further inhibited by this hydrodynamic effect, in addition to the chemical effect.

This conclusion is further elucidated in the PAH-based growth source term analysis, shown in Fig. 7. The legend values in the hydrogen diluted case are reduced by a factor of six to account for the predicted chemical effects on soot formation due to PAH reduction. Therefore, a direct color comparison better demonstrates any additional hydrodynamic effects on the source term. The color pattern near the central jet exit matches between two flames, indicating that chemical effects are dominant close to the jet exit. Conversely, the source term decrease is even more drastic, with

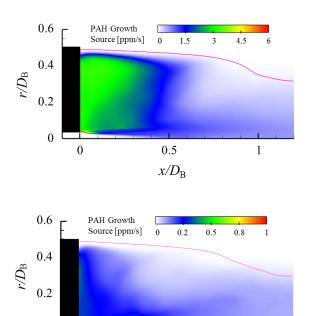


Figure 7: Time-averaged soot volume fraction source term [ppm/s], including both PAH dimer nucleation and condensation, in the recirculation zone of ethylene (top) and ethylene/hydrogen (bottom) flames. Note that the color legend of the ethylene/hydrogen flame is scaled down by six times. The magenta line correspond to the stoichiometric mixture fraction contour.

0.5 $x/D_{\rm B}$

0

0

the chemical effects already being accounted, between $r/D_{\rm B}=0.1$ and 0.4, which agrees with the decrease in mixture fraction in Fig. 6. This again demonstrates the additional hydrodynamic effects on soot reduction due to increased jet velocity.

5. Conclusions

A sooting turbulent bluff body stabilized ethylene/hydrogen flame was studied both experimentally and computationally and compared with a previously analyzed neat ethylene counterpart [11]. Laser-induced incandescence (LII) was utilized to measure the soot volume fraction in the flame. A integrated Large Eddy Simulation (LES) model was adopted to elucidate the interactions between soot, turbulence, and chemistry. The statistical soot model was based on the Hybrid Method of Moments (HMOM), considering detailed nucleation, condensation, particle collision, surface growth, oxidation, and fragmentation processes that influence soot evolution. The combustion model for the gas-phase was based on the Radiation Flamelet/Progress Variable (RFPV) model with modifications to account for the removal of Polycyclic Aromatic Hydrocarbon (PAH) from the gas-phase. A presumed PDF was utilized to model unresolved subfilter scale interactions.

Similar to the ethylene bluff body flame, three distinct flow regions were observed experimentally: a sooting recirculation zone, a non-sooting, high strained neck region, and a sooting jet-like zone. Although the hydrogen diluted

bluff body flame is significantly less sooting than the ethylene counterpart overall, soot reduction in the recirculation zone near the bluff body is more pronounced compared to the downstream jet-like region. Both chemical and hydrodynamic effects were identified as reasons for this decrease.

The chemical effect was benchmarked from a slighted strained steady flamelet calculation. Due to the hydrogen addition, PAH mass fraction was found to decrease by a factor of 2.5, indicating a factor of six decrease in the PAH-based growth rate that includes both PAH dimer nucleation and condensation. This agrees with the soot reduction rate in the downstream jet-like region, where PAH-based growth is dominant. Therefore, the chemical effect is dominant in this region.

Conversely, both the experimental measurement and soot production source term analysis in LES demonstrated additional soot reduction in the recirculation zone, which could not be explained based on the chemical effect alone. In the experiment, to remain the same Reynolds number in the hydrogen diluted flame as the ethylene flame, the central jet velocity was increased. This increase in the fuel to air coflow momentum flux ratio allows relatively less fuel and more air in the recirculation zone near the bluff body surface, compared to the ethylene flame. As a consequence, soot formation is inhibited due to leaner mixture fraction in the recirculation zone. This hydrodynamic effect together with the chemical effect accounts for the soot reduction in the recirculation zone.

Acknowledgments

Please check with our Australian coauthors.

References

- [1] H. Wang, Proc. Combust. Inst. 33 (2011) 41-67.
- [2] C. S. Yoo, H. G. Im, Proc. Combust. Inst. 31 (2007) 701–708.
- [3] D. O. Lignell, J. H. Chen, P. J. Smith, T. Lu, C. K. Law, Combust. Flame 151 (2007) 2-28.
- [4] D. O. Lignell, J. H. Chen, P. J. Smith, Combust. Flame 155 (2008) 316-333.
- [5] F. Bisetti, G. Blanquart, M. E. Mueller, H. Pitsch, Combust. Flame 159 (2012) 317–335.
- [6] A. Attili, F. Bisetti, M. E. Mueller, H. Pitsch, Combust. Flame 161 (2014) 1849–1865.
- [7] A. Attili, F. Bisetti, M. E. Mueller, H. Pitsch, Proc. Combust. Inst. 35 (2015) 1215–1223.
- [8] M. E. Mueller, H. Pitsch, Combust. Flame 159 (2012) 2166–2180.
- [9] C. R. Shaddix, J. Zhang, 8th U.S. National Combustion Meeting, 2013, 070LT-0100.
- [10] Y. Xuan, G. Blanquart, Proc. Combust. Inst. 35 (2015) 1911–1919.
- [11] M. E. Mueller, Q. N. Chan, N. H. Qamar, B. Dally, H. Pitsch, Combust. Flame 160 (2013) 1298–1309.
- [12] P. A. Tesner, Proc. Combust. Inst. 7 (1958) 546–553.
- [13] P. Dearden, R. Long, J. Appl. Chem. 18 (1968) 243-251.
- [14] D. X. Du, R. L. Axelbaum, C. K. Law, Comb. Flame 102 (1995) 11–20.
- [15] O. L. Gülder, D. R. Snelling, R. A. Sawchuk, Proc. Combust. Inst. 26 (1996) 2351–2358.
- [16] H. Guo, F. Liu, G. J. Smallwood, O. L. Gülder, Combust. Flame 145 (2006) 324-338.
- [17] H. Zhao, R. Stone, B. Williams, Energy Fuels 28 (2014) 2144–2151.

- [18] B. B. Dally, A. R. Masri, R. S. Barlow, G. J. Fiechtner, D. F. Fletcher, Proc. Combust. Inst. 26 (1996) 2191–2197.
- [19] B. B. Dally, D. F. Fletcher, A. R. Masri, Combust. Theor. Model. 2 (1998) 193-219.
- [20] N. H. Qamar, Z. T. Alwahabi, Q. N. Chan, G. J. Nathan, D. Roekaerts, K. D. King, Combust. Flame 156 (2009) 1339-1347.
- [21] C. Schulz, B. F. Kock, M. Hofmann, H. Michelsen, S. Will, B. Bougie, R. Suntz, G. Smallwood, Appl. Phys. B 83 (2006) 333–354.
- [22] H. Bladh, P. E. Johnsson, J. Bengtsson, Appl. Phys. B 90 (2008) 109–125.
- [23] C. A. Schuetz, M. Frenklach, Proc. Combust. Inst. 29 (2002) 2307–2314.
- [24] D. Wong, C. A. Schuetz, M. Frenklach, in: H. Bockhorn, A. D'Anna, A. Sarofim, H. Wang (Eds.), Combustion Generated fine Carbonaceous Particles, KIT Scientific Publishing, 2009, pp. 247–257.
- [25] G. Blanquart, H. Pitsch, in: H. Bockhorn, A. D'Anna, A. Sarofim, H. Wang (Eds.), Combustion Generated fine Carbonaceous Particles, KIT Scientific Publishing, 2009, pp. 439–466.
- [26] M. E. Mueller, G. Blanquart, H. Pitsch, Combust. Flame 156 (2009) 1143–1155.
- [27] M. Frenklach, H. Wang, Proc. Combust. Inst. 23 (1991) 1559-1566.
- [28] A. Kazakov, H. Wang, M. Frenklach, Combust. Flame 100 (1995) 111-120.
- [29] K. G. Neoh, J. B. Howard, A. F. Sarofim, in: Particulate Carbon Formation during Combustion, Plenum Press, 1981, pp. 162–282.
- [30] M. E. Mueller, G. Blanquart, H. Pitsch, Proc. Combust. Inst. 33 (2011) 667-674.
- [31] M. Ihme, H. Pitsch, Phys. Fluids 20 (2008) 055110.
- [32] M. E. Mueller, H. Pitsh, Phys. Fluids 23 (2011) 115104.
- [33] A. W. Cook, J. J. Riley, Phys. Fluids 6 (1994) 2868-2870.
- [34] M. Ihme, H. Pitsch, Combust. Flame 155 (2008) 90-107.
- [35] M. Germano, U. Piomelli, P. Moin, W. H. Cabot, Phys. Fluids A 3 (1991) 1760–1765.
- [36] C. Meneveau, T. S. Lund, W. H. Cabot, J. Fluid Mech. 319 (1996) 353-385.
- [37] H. Pitsch, FlameMaster, a C++ computer program for 0D combustion and 1D laminar flame calculations.
- [38] G. Blanquart, P. Pepiot, H. Pitsch, Combust. Flame 156 (2009) 588-607.
- [39] K. Narayanaswamy, G. Blanquart, H. Pitsch, Combust. Flame 157 (2010) 1879–1898.
- $[40]\ \ O.\ Desjardins,\ G.\ Blanquart,\ G.\ Balarac,\ H.\ Pitsch,\ J.\ Comput.\ Phys.\ 227\ (2008)\ 7125-7159.$
- [41] M. Herrmann, G. Blanquart, V. Raman, AIAA J. 44 (2006) 2879–2886.
- [42] B. B. Dally, A. R. Masri, R. S. Barlow, G. J. Fiechtner, Combust. Flame 114 (1998) 119-148.