# ME/AE 5212 Introduction to Finite Element Analysis



# **Tutorial**

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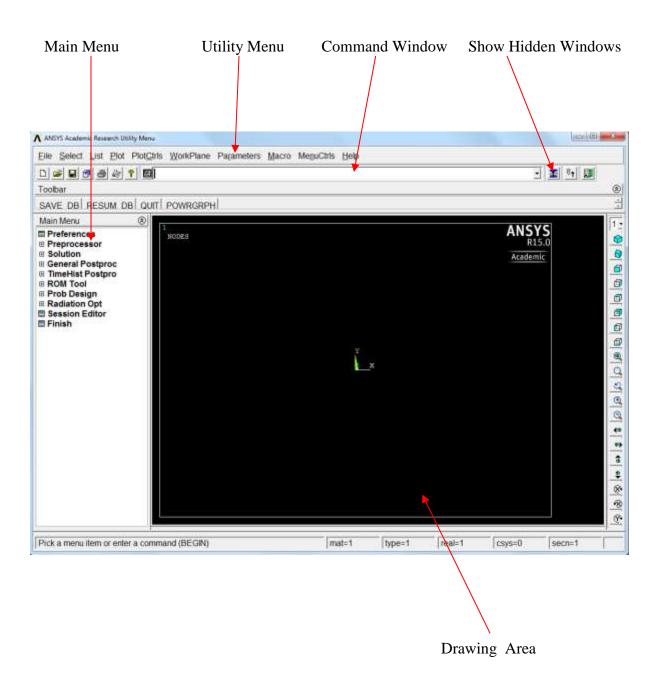


## **ANSYS TUTORIAL**

- GETTING STARTED WITH ANSYS 15.0
- SOLID MECHANICS
  - **1.1. BEAM**
  - 1.2. CIRCULAR PLATE
- HEAT TRANSFER
- FLUID MECHANICS

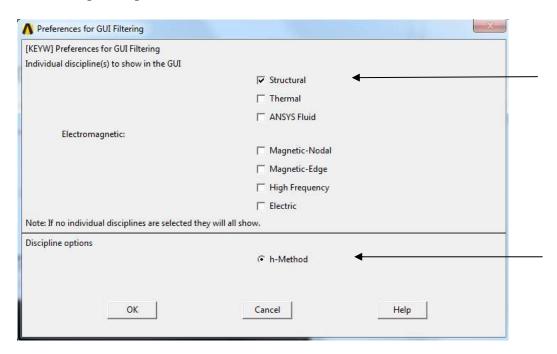
# **INTRODUCTION TO ANSYS 15.0**

In a windows machine start ANSYS by clicking on



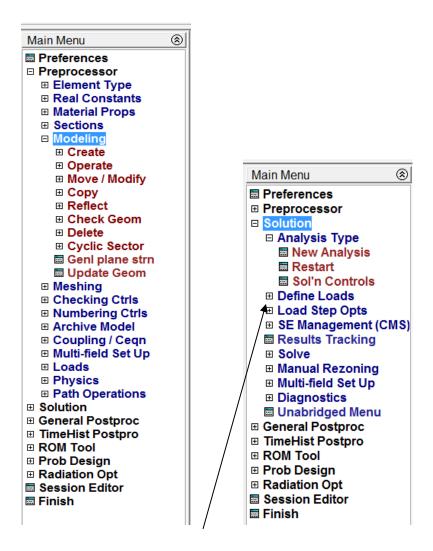
# **GETTING START WITH ANSYS 15.0**

**Step1:** Click on Preferences and choose the type of problem (Structural in this case). Leave the Discipline Options as default (h-method).



**Step 2:** Enter Preprocessor to establish the Finite Element Model.

- There are two approaches to build a FE model
  - o From Top To Bottom
  - o From Bottom to Top
- The general steps to follow in the analysis are:
  - 1. Choose the Element Type for the Analysis
  - 2. Set the Real constant Values for the element (if necessary)
  - 3. Define the Material Properties (Modulus, Poisson's Ratio etc)
  - 4. Create the Finite Element Model using the available tools in the create menu



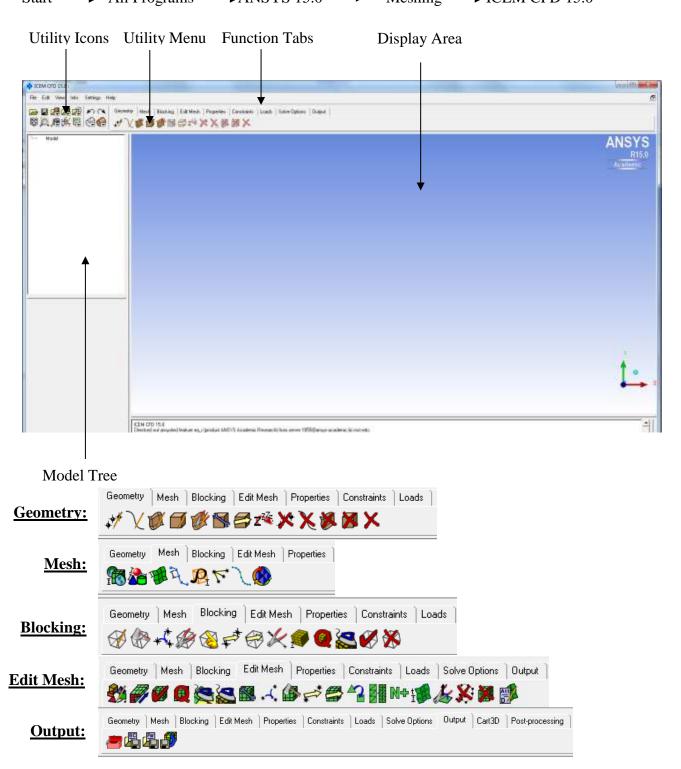
**Step 3:** Enter Solution Menu to Apply the Boundary Conditions and Loads and then solve the Problem

**Step 4:** Once the solution is done, enter the General Postprocessor to List and View the Results



In this manual, ICEM (Integrated Computer-Aided Engineering and Manufacturing) is used to create and mesh the computational domain for fluid mechanics problems. After creation of the mesh, ANSYS CFX will be used to solve the flow problem and analyze the solution.

Start → All Programs → ANSYS 15.0 → Meshing → ICEM CFD 15.0

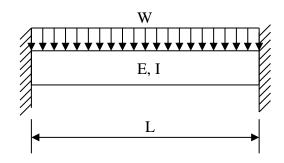


# **SOLID MECHANICS**

### 1.1 Analysis of a Beam

Determine the mid-span deflection and reactions for the fixed-fixed beam subjected to a uniformly distributed load as shown in the figure.

Take W=500 N/m, E = 200GPa, Poisson's ratio=0.3, L = 10 m, B=H=0.124467m, I= $2x \ 10^{-5} \ m^4$ 



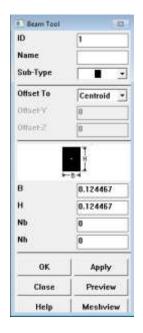
### STEP 1:

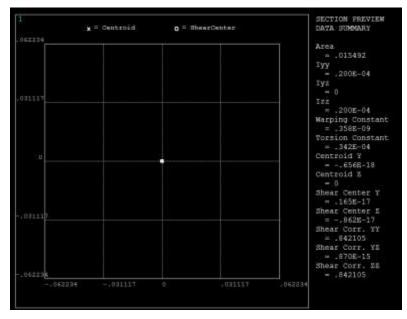
Preferences → Structural → OK

### **STEP 2:** Define Element Type

### **STEP 3:** Define the Section

Preprocessor → Sections → Beam → Common Sections → Define the section as figure below → Apply → Preview, the figure below pop up





### **STEP 4: Define Material Properties**

Enter EX = 200e9 and Poisson's Ratio =  $0.3 \rightarrow OK$ 

### **STEP 5:** Create the Keypoints

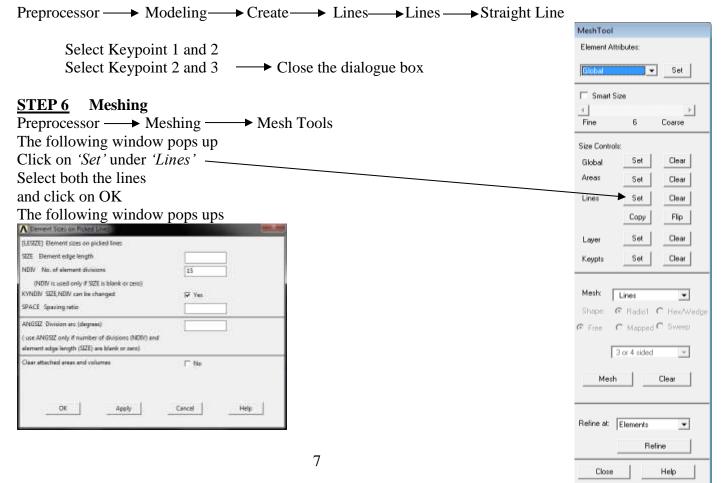
### Create keypoints:

Enter Keypoint number and X, Y, Z Location in active CS

Keypoint Number	X	Y	Z	
1	0.0	0.0	0.0	→ Apply
2	5.0	0.0	0.0	→ Apply
3	10.0	0.0	0.0	<b>→</b> OK

The Keypoint numbers appear on the Graphics Window.





Element Attributes: Set number of element divisions to 15 and click on OK ☐ Smart Size The display on the graphical interface changes to Fine Size Controls: Set Clear Global Set Clear Set Clear Сору Flip Set Clear Now click on 'Mesh' in the 'Mesh Tool' window Lines Select both the lines from the graphical interface. C Mapped C Sweep Click OK Display the element and node numbers. 3 or 4 sided  $\, \forall \,$ Utility menu → PlotCtrls → Numbering Toggle on the 'Node Numbers' option. (If the drawing is disappeared, please go to, Plot Refine at: Elements (from utility menu on the top)  $\longrightarrow$  Multi-Plots)  $\longrightarrow$  OK Close Help 6 7 8 9 10 11 12 13 14 15 16 2 18 19 20 21 22 23 24 25 26 27 28 29 30 31 17

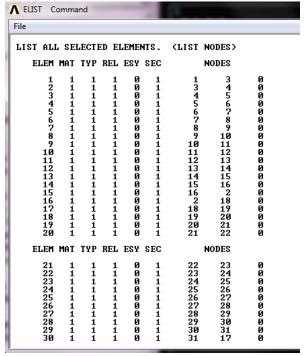
MeshTool

After finishing this step, go to the List to check the element, nodes material assignment.

Utility menu 

List 

Rodes+Attr+Realconstant



**STEP 7:** Apply Boundary Conditions and Loads

### Apply B.C's

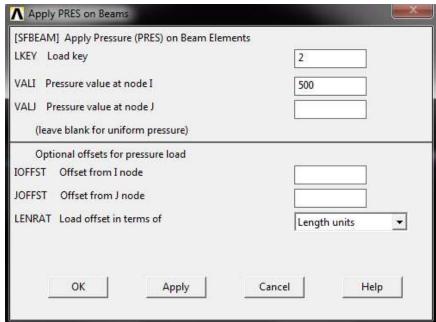
Solution — Define loads — Apply — Structural — Displacement — On Nodes. Select the two end nodes (click on nodes at the most left and right) — OK (or middle click).

Select ALL DOF in the dialog box appeared → OK.

The Boundary conditions will be displayed on the graphics window.

### Apply Pressure on Beam

Solution → Define loads → Apply → Structural → Pressure → On Beams Use the Box method to select all the elements and middle click. The following dialog box appears:



Enter the value of Load key=2, pressure = 500 and click OK. The pressure will be displayed on the graphics window.

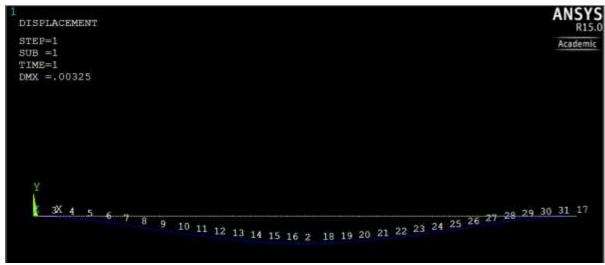


### **STEP 8:** Solve the problem

Solution  $\longrightarrow$  Solve  $\longrightarrow$  Current LS. Click OK to solve.

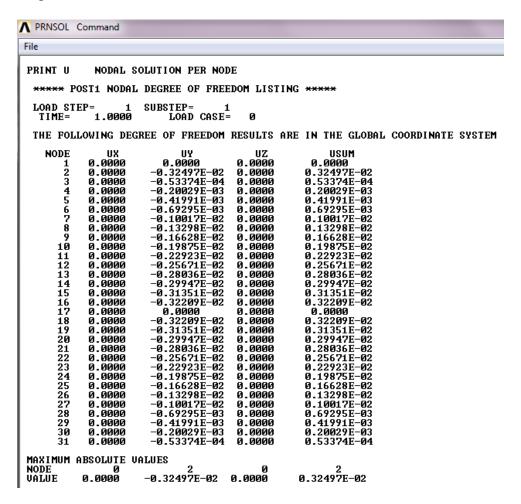
#### **STEP 9:** View results.

General Postproc  $\longrightarrow$  Plot Results  $\longrightarrow$  Deformed Shape Select Def + undeformed and select OK.



#### **Nodal Solution**

General Postproc → List Results → Nodal Solution → DOF solution → Displacement vector sum → OK



#### The displacement at node 2 is -0.0032497 m in the Y direction.

### **Reaction Solution**

General Postproc → List Results → Reaction Solution → All items → OK

```
File

PRINT REACTION SOLUTIONS PER NODE

***** POST1 TOTAL REACTION SOLUTION LISTING *****

LOAD STEP= 1 SUBSTEP= 1
    TIME= 1.0000 LOAD CASE= 0

THE FOLLOWING X,Y,Z SOLUTIONS ARE IN THE GLOBAL COORDINATE SYSTEM

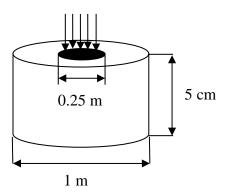
NODE FX FY FZ MX MY MZ
1 0.0000 2500.0 0.0000 0.20840E-13 0.25871E-12 4162.0
17 0.0000 2500.0 0.0000 0.20840E-13-0.25871E-12 -4162.0

TOTAL VALUES
UALUE 0.0000 5000.0 0.0000 0.41679E-13 0.11117E-24 0.19536E-08
```

<u>Note</u>: Due to variances of computation rounding, the final results could be a little different from the results above.

### 1.2 Analysis of a Circular Plate Using Axisymmetric Element

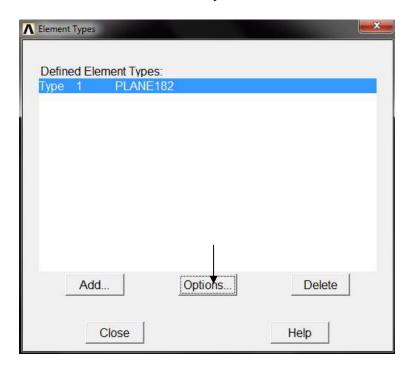
A circular aluminum plate with 1m diameter and 5cm thickness is loaded by a uniform pressure of 0.5Pa distributed over a 0.25m diameter circle at the center of the plate. The outer edges of the plate are completely fixed from motion. Find the maximum deflection and von Mises stress. Use Young's modulus = 70GPa, Possion's ratio=0.33.

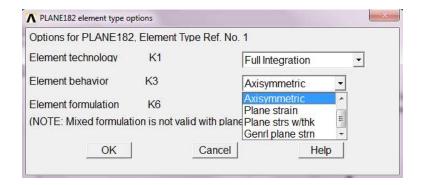


<u>STEP 1:</u>
Preferences → Structural → OK

### **STEP 2:** Define Element Type

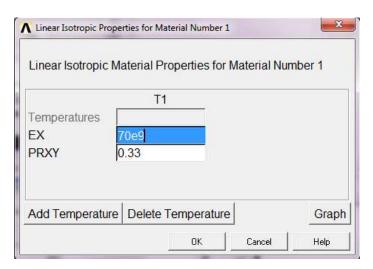
Preprocessor — Element Type — Add/Edit/Delete — Add... — Solid — Quad 4 node 182 — OK — Close *Element Types* dialog box — Options — Element Behavior K3: Axisymmetric — OK





### **STEP 3:** Define Material Properties

Preprocessor → Material Props → Material Models → Structural → Linear → Elastic → Isotropic



Enter Young's Modulus (EX) and Poisson's ratio (PRXY) for the material and click OK.

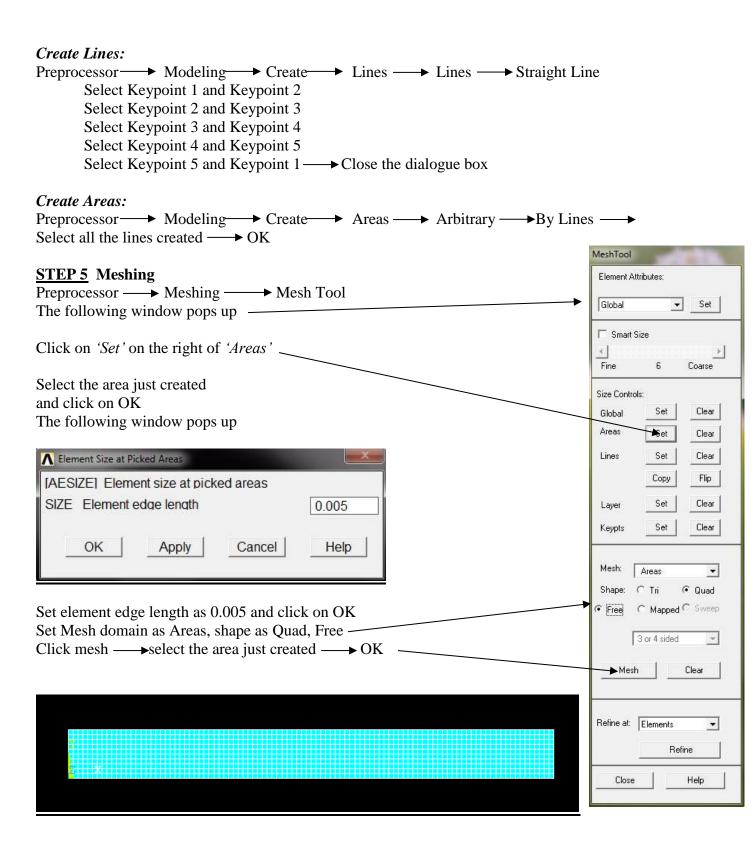
### **STEP 4:** Create the model

### Create keypoints:

Enter Keypoint number and X, Y, Z Location in active CS

Keypoint Number	X	Y	Z	
1	0.0	0.0	0.0	→ Apply
2	0.5	0.0	0.0	→ Apply
3	0.5	0.05	0.0	→ Apply
4	0.125	0.05	0.0	→ Apply
5	0.0	0.05	0.0	→ OK

The Keypoint numbers appear on the Graphics Window.



After finishing this step, go to the List to check the element, nodes and material assignment.

Utility menu 

List 

Nodes+Attr+Realconstant

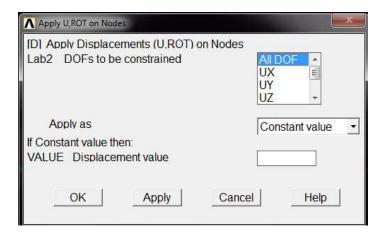
↑ ELIST Command										
File										
LIST ALL SELECTED ELEMENTS. (LIST NODES)										
ELEM MAT TYP REL ESY SEC						NODES				
	1	1	1	1	0	1	1111	1110	1017	1018
	2	1 1 1	1	1	0	1	1018	1019	1020	1111
	3	1		1 1 1	0 0		1111	834	835	1110
	1 2 3 4 5 6 7 8 9	1 1 1	1 1 1 1	1	0 0	1 1 1 1	1111	1020	833	834
	5	1	1	1	0	1	835	836	1109	1110
	6	1	1	1	0	1	836	837	1108	1109
	7	1 1 1	1	1	0 0	1	1109	1016	1017	1110
	8	1	1	1	0	1	1108	1015	1016	1109
	9	1	1 1	1	0	1 1	837	838	1107	1108
	10	1	1	1	0 0 0 0	1	1107	1014	1015	1108
	11 12	1	1 1 1	1	0	1	838	839	1106	1107
	12	1	1	1	0	1 1 1	1106	1013	1014	1107
	13	1	1	1	0	1	839	840	1105	1106
	14	1	1	1	0	1	1105	1012	1013	1106
	15	1 1 1 1 1 1 1 1 1	1 1	1	0	1	840	841	1104	1105
	16 17	1	1	1	0	1	1104	1011	1012	1105
	17	1	1	1	0 0	1 1	841	842	1103	1104
	18	1	1 1 1	1	Ō	1	1103	1010	1011	1104
	19	1	1	1111111111111111	0	ī	842	843	1102	1103
	20	ī	1	ī	Ō	1 1	1102	1009	1010	1103

**STEP 6:** Apply Boundary Conditions and Loads

### Apply B.C's

Main Menu → Solution → Define loads → Apply → Structural → Displacement → On Lines.
Select the right line → OK (or middle click).

The following dialog box appears:



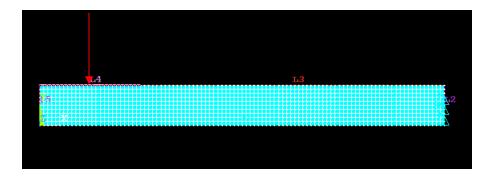
Select ALL DOF and click OK.

The Boundary conditions are displayed on the graphics window.

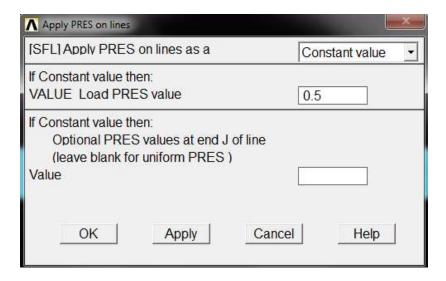
### Apply Pressure

Solution → Define loads → Apply → Structural → Pressure → On Lines.

Select the left line on the top and click *OK*.

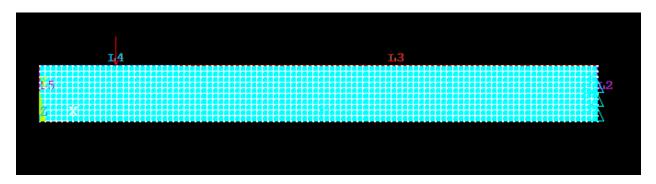


The following dialog box appears:



Set Load PRES value as  $0.5\ \mbox{and click OK}.$ 

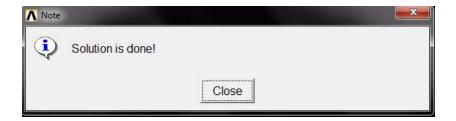
The Graphic Display should look like the figure below.



### **STEP 7:** Solve the problem

Solution → Solve → Current LS.

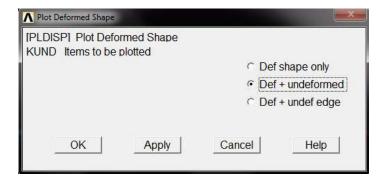
A dialog box appears. Click OK to solve. Once the solution is done, the following dialog box appears:-



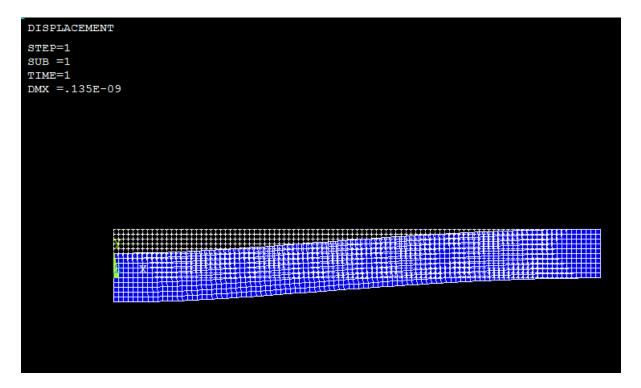
Close the dialog box.

<u>STEP 8:</u> View results.

Main Menu → General Postproc → Plot Results → Deformed Shape The following dialog box appears:



Select Def + undeformed and select OK.



### **Nodal Solution**

General Postproc 
List Results 
Nodal Solution 
DOF Solution

Y-Component of displacement

\*\*\*\*\*\* Post1 Nodal degree of freedom listing \*\*\*\*\*

Load Step= 1 Substep= 1
Time= 1.0000 Load case= 0

The following degree of freedom results are in the global coordinate system

Node UY
1111 -0.13419E-09

MAXIMUM ABSOLUTE VALUES
NODE 7 VALUE -0.13533E-09

The maximum displacement in Y direction happens at node 7, the value is 0.13533E-9 m. Note: The node numbers in your model will depend upon the model building and meshing sequence you use and may be different from what is shown here.

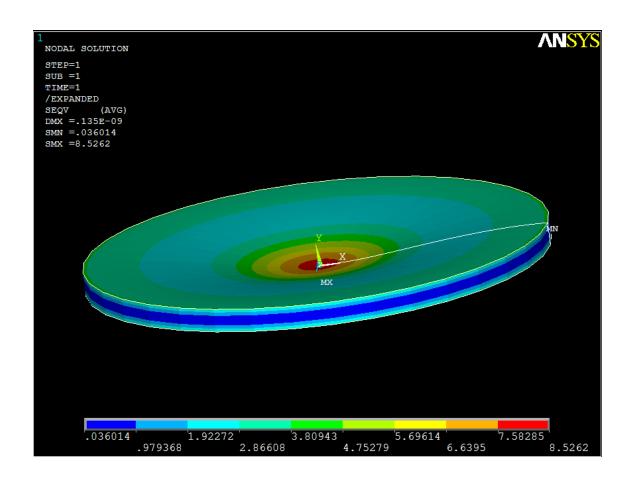
General Postproc → List Results → Nodal Solution → Stress → von Mises stress

1109 0.13722 1110 0.10430 -0.46488E-01-0.43371 -0.46465E-01-0.40074 0.57093 0.50504 \*\*\*\*\* POST1 NODAL STRESS LISTING \*\*\*\*\*
PowerGraphics Is Currently Enabled LOAD STEP= 1 SUBSTEP= 1 TIME= 1.0000 LOAD CASE= 0 NODAL RESULTS ARE FOR MATERIAL 1 NODE S1 S2 S3 1111 0.72579E-01-0.46449E-01-0.36899 MINIMUM VALUES NODE 113 VALUE -1.2504 225 225| 0.41537E-01 0.36014E-01 -8.2071 -8.3310 MAXIMUM VALUES NODE VALUE 8.5262 8.1137 8.5877 8.2382

### The maximum von Mises stress happens at node 2, the value is 8.5262 pa.

To view the node number, go to Utility menu — PlotCtrls — Numbering — Toggle on the 'Node Numbers' option — OK (If the drawing disappeared, please go to, Plot (from utility menu on the top) Multi-Plots)

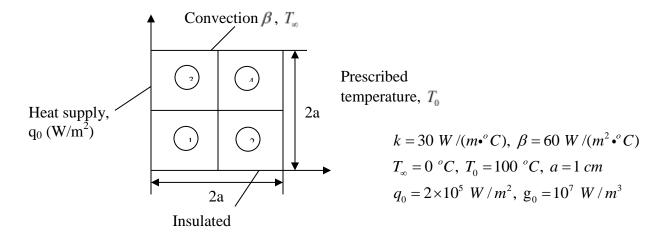
To view the full model, go to Utility menu → PlotCtrls → style → Symmetry Expansion → 2D Axi-symmetric → Full expansion → OK



# **HEAT TRANSFER ANALYSIS**

### Problem 8.30 (Text)

Consider steady-state heat conduction in a square region of side 2a. Assume that the medium has conductivity of k (unit:  $W/(m \cdot {}^{o}C)$ ) and uniform heat (energy) generation of  $g_0$  (unit:  $W/m^3$ ). Plot the contour of temperature distribution and list the nodal temperatures.

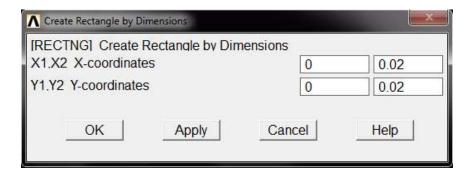


# STEP 1: Preferences → Thermal → OK

### **STEP 2: Define Element Type**

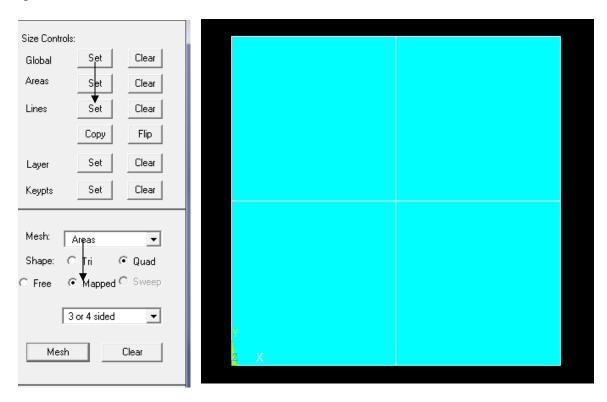
### **STEP 3:** Define Material Properties

#### **STEP 4: Create the Model**



### **STEP 5:** Mesh the Model

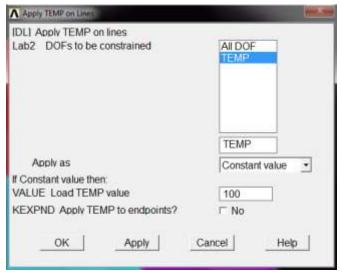
Preprocessor  $\longrightarrow$  Meshing  $\longrightarrow$  Mesh Tool  $\longrightarrow$  Lines  $\longrightarrow$  Set  $\longrightarrow$  Click the four edges of the square  $\longrightarrow$  OK  $\longrightarrow$  NDVI  $\longrightarrow$  input 2  $\longrightarrow$  OK  $\longrightarrow$  Click "Mapped"  $\longrightarrow$  Mesh  $\longrightarrow$  Click the Square  $\longrightarrow$  OK  $\longrightarrow$  Close



**STEP 6:** Apply Boundary Conditions and Loads

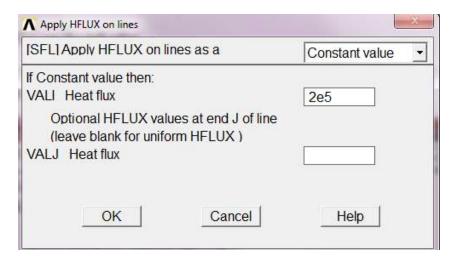
### Apply temperature on the right edge

Preprocessor Loads Define Loads Apply Thermal Temperautre
On Lines Click the right edge of the square OK Input the following values
OK



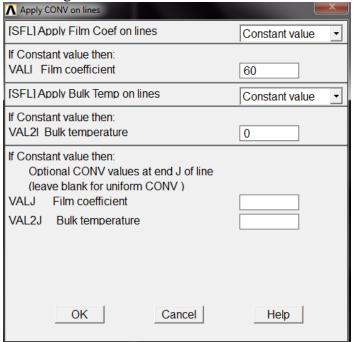
### Apply Heat Flux on the left edge

Apply → Thermal → Heat Flux → On Lines → Click the left edge of the square → OK → Input the following values → OK



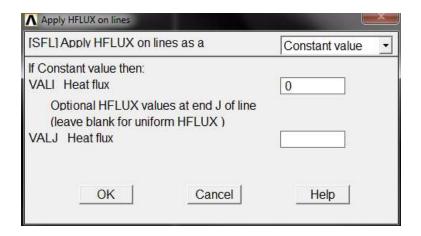
### Apply Convection on the top edge

Apply → Thermal → Convection → On Lines → Click the top edge of the square → OK → Input the following values → OK



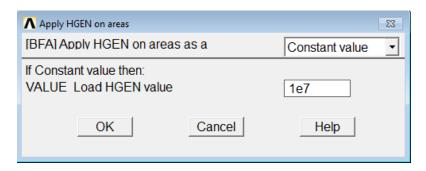
### Apply Insulated B.C. on the bottom edge

Apply → Thermal → Heat Flux → On Lines → Click the bottom edge of the square → OK → Input the following values → OK



### Apply Volume heat generation over the whole modeling

Apply → Thermal → Heat Generat → On Areas → Select the area created → OK (or middle click) → Input the following values → OK

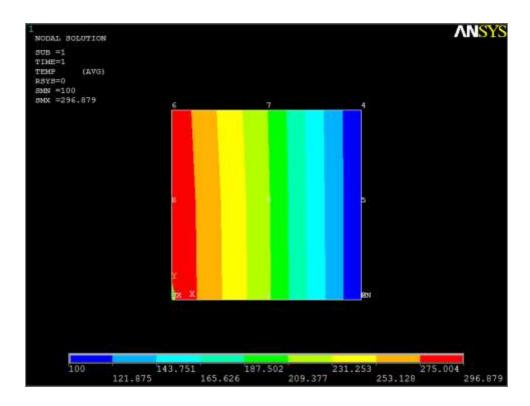


### **STEP 7:** Solve the problem

Solution  $\longrightarrow$  Solve  $\longrightarrow$  Current LS. Click OK to solve.

### **STEP 8:** View results.

General Postproc → Plot Results → Contour Plot → Nodal Solu → DOF Solution → Nodal Temperature → OK



### View the temperature value at all nodes

```
Main Menu → Plot → Mult-plots

Main Menu → Numbering → Node Numbers → Click "On" → OK

General Postproc → List Results → Nodal Solution → DOF Solution → Nodal

Temperature → OK
```

```
File

PRINT TEMP NODAL SOLUTION PER NODE

******* POST1 NODAL DEGREE OF FREEDOM LISTING ******

LOAD STEP= 0 SUBSTEP= 1
TIME= 1.0000 LOAD CASE= 0

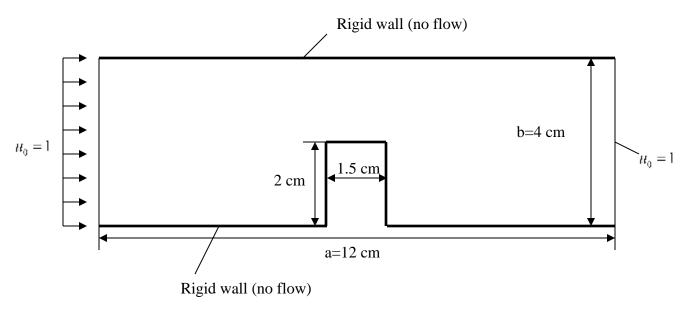
NODE TEMP
1 296.88
2 100.00
3 214.46
4 100.00
5 100.00
6 291.55
7 210.63
8 295.74
9 213.66 |

MAXIMUM ABSOLUTE VALUES
NODE 1
UALUE 296.88
```

# **FLUID MECHANICS**

### Problem 8.42 (Text)

For the fluid problem shown below, determine the steady velocity distribution of the domain

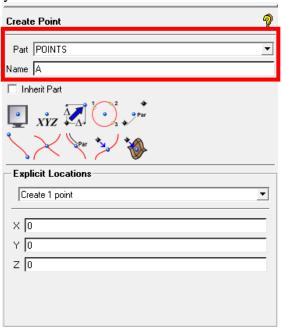


### **STEP 1:** Open ICEM CFD & Specify file name & its path

Start → Program → ANSYS 15.0 → Meshing → ICEM CFD 15.0 → File → New Project → Browse to a working directory → Save

### **STEP 2:** Create the vertices to create lines and the surface of the domain

Geometry (function tab) → Create Point → Explicit Coordinates → Enter the X, Y, Z coordinates → Apply



Before creating a point, do not forget to change part name to POINTS as show above and keep the name unchanged during these points' creation.

<u>Note</u>: If your version of ICEM doesn't display the same create point options as above, adjust the following settings.

- 1. Settings → Selection → Auto pick mode should be turned OFF.
- 2. Settings → Geometry Options → Name new geometry must be turned ON.
- 3. Settings → Geometry Options → Replace same name item must be turned ON.
- 4. Settings → Geometry Options → Inherit Part Name → Create new must be toggled ON.
- 5. Settings → Geometry Options ← Create surface topology must be turned OFF.

Create the following set of points with the names given below:

NAME	A	В	C	D	Е	F	G	Н
X	0	0.0525	0.0525	0.0675	0.0675	0.12	0.12	0
Y	0	0	0.02	0.02	0	0	0.04	0.04
Z	0	0	0	0	0	0	0	0

In the model tree, Geometry → (right click) Points → Show Points Names, all the points' name created could be showed, and make sure no duplication of same points.

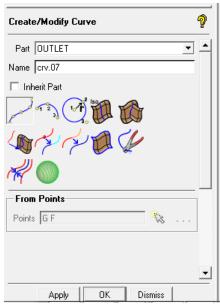
#### **STEP 3:** Create lines from the vertices created

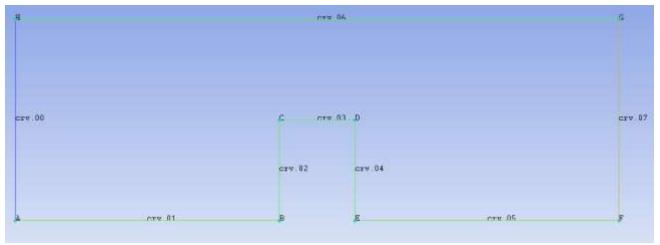
Geometry (function tab) → Create/modify curve → From points → Change part name to INLET → At the blank Points, fill in A H → APLLY (You can also select points A & H → MIDDEL=DONE)

Similarly, change part name to WALL —> select AB, middle click —> select BC, middle click —> select CD, middle click —> select EF, middle click —> select HG, middle click —> Dismiss, all these lines are belonged to part WALL.

Note: Do not change part name when creating WALL lines.

In the model tree, right click on Geometry — Curves — Show Points Names, all the curves' name created could be showed, and make sure no duplication of same curve. Similarly, join GF, and name this line as OUTLET.





<u>Note:</u> After every step is done, click File — Save Project to make sure the changes made are saved in time.

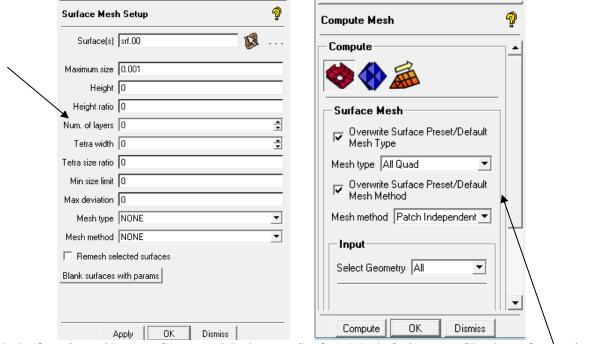
### **STEP 4:** Create the surface

Geometry (function tab) — Create/modify surface — Change Part Name to SURFACE — Simple surface — Methods — From curves — Select all the lines created — Middle click — APPLY — Dismiss.

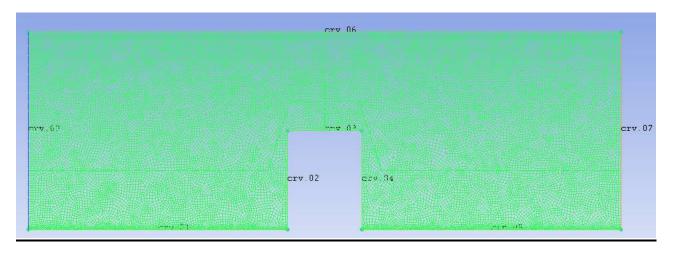


### **STEP 5:** Surface meshing

Mesh (function tab) → Surface Mesh Setup → Surface → Select the SURFACE created → Middle click → Maximum size → 0.001 → OK.



Mesh (function tab) → Compute Mesh → Surface Mesh Only → Check on Overwrite Surface Preset/Default Mesh Type → Mesh Type → All Quad → Check on Overwrite Surface Preset/Default Mesh Method → select Patch Independent → Select Geometry → ALL → Compute.

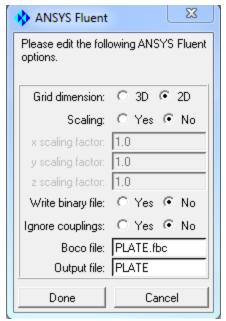


#### **STEP 6:** Select the solver

Output (function tab) → Select solver → Output Solver=ANSYS Fluent → Common Structural Solver=ANSYS → OK

### **STEP 7:** Saving the mesh file

Output (function tab) → Write input → save project first=YES → Open the .uns file → Save the file in 2D → Change output file name to PLATE → DONE.



The extension of the mesh file would be .msh, which can be read by most of the CFD solvers including ANSYS CFX, Star CD, OpenFOAM and FLUENT.

### **STEP 8:** Import mesh file into CFX

File 
$$\longrightarrow$$
 New case  $\longrightarrow$  General  $\longrightarrow$  OK

#### STEP 9: Set up steady-state case definition

In the model tree, double click **Analysis Type**  $\longrightarrow$  select steady state  $\longrightarrow$  OK

In the model tree, double click **Default Domain**,

Basic setting (tab) → Change material to Air at 25 degree C → Ref. Pressure= 1 atm.

Fluid Models (tab) → turbulence → (none) laminar → Combustion → None → OK.

### **STEP 10:** Define boundary conditions

Enter the following:

Option=Cart. Vel. Components  $\longrightarrow$  U=1 m/s, V=W=0 m/s.

In the model tree, under **Simulation** → **Flow analysis** → right click Default domain → Insert → Boundary → Type OUTLET → OK

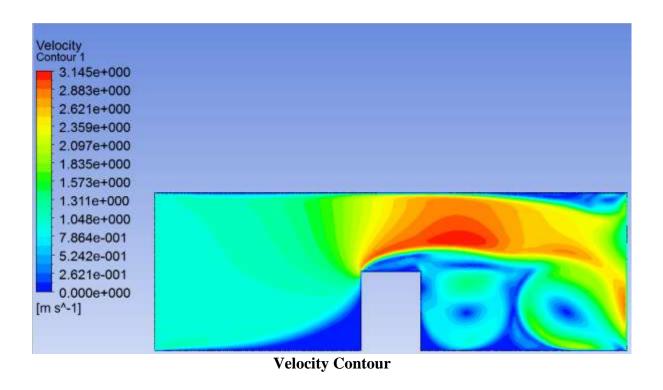
Enter the following:

Basic settings (Tab) → Boundary Type=outlet → Location=OUTLET

Boundary details (Tab) → Flow Regime = Subsonic → Mass & Momentum →

Option=Cart. Vel. Components —→U=1 m/s, V=W=0 m/s.

In the model tree, under **Simulation** → **Flow analysis** → right click Default domain → Insert  $\longrightarrow$  Boundary  $\longrightarrow$  Type symmetry  $\longrightarrow$  OK Enter the following: Basic settings (Tab) → Boundary Type=symmetry → Location=Primitive 2D C, Primitive 2D D (use 'Shift' key)  $\longrightarrow$  OK In the model tree, under **Simulation** → **Flow analysis** → right click Default domain → Insert  $\longrightarrow$  Boundary  $\longrightarrow$  Type wall  $\longrightarrow$  OK Enter the following: Basic settings (Tab) → Boundary Type=wall → Location= wall Boundary details (Tab) → Mass & Momentum → Option=No Slip Wall → Apply OK. **STEP 11: Set up solver controls** In the model tree, under **Simulation** → **Flow analysis** → Solver → Double Click Solver Control. Advection Scheme → High resolution Minimum iterations  $\longrightarrow$  1 Maximum iterations → 300 Convergence criteria: Residual type → RMS Residual target → 1E-4 → OK STEP 12: Run Right click on the drawing window → Start solver → Define Run → Save → Start Run. STEP 13: Post processing of results. Menu (On CFX-Solver Manager) → Launch CFD-Post with a Results File → OK. Menu → Insert → Contour → Name: Contour 1. Geometry (Tab) → Location=symmetry → Variable=velocity → Number of contours  $\longrightarrow$  25  $\longrightarrow$  Apply.



Note: The results will vary slightly depending on the mesh size and element type