

# 3D Space Truss Analysis Vector Example

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## 1 3D Truss Analysis Example

This document demonstrates using Jupyter for vector operations by determining the axial forces in a simple 3D space truss.

### 1.1 Initialization

The code below imports the required modules into the notebook. We use NumPy for its arrays. The numpy.linalg gives us `norm` to determine a vectors magnitude. The `set_printoptions` sets the array default printing format. I use X=0, Y=1, Z=2 as index values to return the components of a vector.

```
In [1]: from numpy import *
        from numpy.linalg import *
        set_printoptions(precision=3, suppress=True)
        X=0; Y=1; Z=2
```

### 1.2 Problem Statement

For the image shown to the right, determine all axial forces as well as all reaction forces. The applied force is  $F = (0, 40kN, 0)$

We define all the points for use later in computing unit vectors. We also define a function to make this a trivial task.

```
In [2]: a=array((1.1, -0.4, 0))
        b=array((1,0,0))
        c=array((0,0,0.6))
        d=array((0,0,-0.4))
        e=array((0,0.8,0))

        Fapp=array((0, 40, 0))

        def  $\lambda$ (a,b):
            return (b-a)/norm(b-a)
```

### 1.3 Joint A

I should have an FBD of joint A here, but I will save that for another update.

In vector form, force equilibrium at joint A gives  $T_{AB}\lambda_{AB} + T_{AC}\lambda_{AC} + T_{AD}\lambda_{AD} + F_{app} = 0$ . We need to calculate the unit vectors.

```
In [3]: lab=λ(a,b)
        lac=λ(a,c)
        lad=λ(a,d)
        print('The unit vectors lab, lac, lad are:', '\n',lab, '\n',lac, '\n', lad)
```

The unit vectors lab, lac, lad are:

```
[-0.243  0.97   0.   ]
[-0.836  0.304  0.456]
[-0.889  0.323 -0.323]
```

We can write the above equilibrium equation in matrix form as  $M \cdot T = -F_{app}$ .

In this case,

$$T = \begin{bmatrix} T_{AB} \\ T_{AC} \\ T_{AD} \end{bmatrix}$$

and  $M$  is composed of the unit vectors as defined in the cell below.

```
In [4]: m=matrix((lab[X],lac[X],lad[X]),
                  (lab[Y],lac[Y],lad[Y]),
                  (lab[Z],lac[Z],lad[Z]))
        print(m)
```

```
[[-0.243 -0.836 -0.889]
 [ 0.97   0.304  0.323]
 [ 0.     0.456 -0.323]]
```

The solution is then:

$$\begin{bmatrix} T_{AB} \\ T_{AC} \\ T_{AD} \end{bmatrix} = M^{-1} \cdot (-F_{app})$$

which we calculate in the following cells.

```
In [5]: minv=inv(m)
        print(minv)
```

```
[[ 0.412  1.134 -0.   ]
 [-0.526 -0.132  1.315]
 [-0.742 -0.186 -1.237]]
```

```
In [6]: answer=minv.dot(-Fapp)
        (TAB, TAC, TAD)=answer.tolist()[0]
        print("The results are TAB={:.3f}, TAC={:.3f} and TAD={:.3f}.".format(TAB,
```

The results are  $TAB=-45.354$ ,  $TAC=5.261$  and  $TAD=7.422$ .

## 1.4 Joint B

We proceed in the exact same way at this joint. The only difference is that the known force comes from TAB, not an externally applied force.

```
In [9]:  $\lambda_{ba}=\lambda(b,a)$ 
         $\lambda_{bc}=\lambda(b,c)$ 
         $\lambda_{bd}=\lambda(b,d)$ 
         $\lambda_{be}=\lambda(b,e)$ 
        print( $\lambda_{ba}$ , '\n',  $\lambda_{bc}$ , '\n',  $\lambda_{bd}$ , '\n',  $\lambda_{be}$ )
```

```
[ 0.243 -0.97   0.   ]
[-0.857  0.    0.514]
[-0.928  0.   -0.371]
[-0.781  0.625  0.   ]
```

```
In [10]: m=matrix(( ( $\lambda_{bc}[X]$ ,  $\lambda_{bd}[X]$ ,  $\lambda_{be}[X]$ ),
                    ( $\lambda_{bc}[Y]$ ,  $\lambda_{bd}[Y]$ ,  $\lambda_{be}[Y]$ ),
                    ( $\lambda_{bc}[Z]$ ,  $\lambda_{bd}[Z]$ ,  $\lambda_{be}[Z]$ )))
        print(m)
```

```
[[-0.857 -0.928 -0.781]
 [ 0.     0.     0.625]
 [ 0.514 -0.371  0.   ]]
```

```
In [11]: minv=inv(m)
        print(minv)

[[-0.466 -0.583  1.166]
 [-0.646 -0.808 -1.077]
 [ 0.     1.601  0.   ]]
```

```
In [12]: FonBfromA=TAB *  $\lambda_{ba}$  #note this is the unit vector from B to A, not A to B.
        print(FonBfromA)
```

```
[-11.  44.  -0.]
```

```
In [13]: answer=minv.dot(-FonBfromA)
        (TBC, TBD, TBE)=answer.tolist()[0]
        print("The results are TBC={:.3f}, TBD={:.3f} and TBE={:.3f}.".format(TBC,
```

The results are  $TBC=20.525$ ,  $TBD=28.434$  and  $TBE=-70.434$ .

## 1.5 Joint E

In this case, the equilibrium equation only has the known axial force from BE and the three reaction forces at E. This makes the equilibrium equation very simple.

```
In [15]:  $\lambda_{eb} = -\lambda_{be}$ 
         m=identity(3)
         print(m)
```

```
[[ 1.  0.  0.]
 [ 0.  1.  0.]
 [ 0.  0.  1.]]
```

```
In [16]: minv=inv(m)
         minv
```

```
Out[16]: array([[ 1.,  0.,  0.],
                [ 0.,  1.,  0.],
                [ 0.,  0.,  1.]])
```

```
In [17]: FonEfromBE= TBE *  $\lambda_{eb}$ 
         print(FonEfromBE)
```

```
[-55.  44.   0.]
```

```
In [19]: E=dot(minv,-FonEfromBE)
         print(E)
```

```
[ 55. -44.   0.]
```

## 1.6 Joint C

```
In [21]:  $\lambda_{ca} = -\lambda_{ac}$ 
          $\lambda_{cb} = -\lambda_{bc}$ 
```

```
In [24]: FonCfromAC=TAC *  $\lambda_{ca}$ 
         FonCfromBC=TBC *  $\lambda_{cb}$ 
         print('The force on C from member AC is {} \n' 'The force on C from member
               .format(FonCfromAC, FonCfromBC))
```

```
The force on C from member AC is [ 4.4 -1.6 -2.4]
```

```
The force on C from member BC is [ 17.6  -0.  -10.56]
```

```
In [26]: C=dot(minv,-(FonCfromAC+FonCfromBC))
         print(C)
```

```
[-22.    1.6   12.96]
```

## 1.7 Joint D

```
In [28]:  $\lambda_{da} = -\lambda_{ad}$   
          $\lambda_{db} = -\lambda_{bd}$ 
```

```
In [29]: D = dot(minv, -(TAD *  $\lambda_{da}$  + TBD *  $\lambda_{db}$ ))  
         print(D)
```

```
[-33.      2.4  -12.96]
```

We can now check our work. The sum of the reaction forces should equal the applied force.

```
In [30]: Freaction = C + D + E  
         print(Freaction)
```

```
[ 0. -40.  0.]
```

This is the negative of the applied force.

```
In [32]: print(Fapp)
```

```
[ 0 40  0]
```