Assignment 1 - Computational Dynamics

Assignment 1 - 4 bar linkage

The goal of this assignment is to perform a full kinematic analysis of a four-bar linkage.

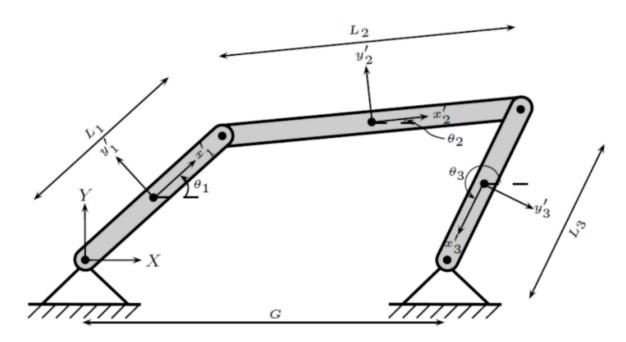


Figure 1: Four-bar linkage

Use the following lengths:

$$L_1 = 10m; L_2 = 26m; L_3 = 18m; G = 20m$$

Problem:

1. Identify the number of bodies, joints and degrees of freedom for the mechanism.

The number of bodies (nb) is 3 and there are 4 joints with 1 degree of freedom each (nh = 8).

Therefore there degrees of freedom for the model is $3 \cdot nb - nh = 1$

The remaining DoF will be governed by a driving constraint $\Phi^D(q,t)=\phi_1-\omega t, \quad \omega=1.5 {{\rm rad} \over {\rm s}}$

2. Setup the kinematic constraints Φ^K for all the joints

First A and r are defined:

$$A(\phi) = \begin{bmatrix} \cos(\phi) & -\sin(\phi) \\ \sin(\phi) & \cos(\phi) \end{bmatrix}, \quad r^p = r + A(\phi) s'^P, \quad q = \begin{bmatrix} x_1 & y_1 & \phi_1 & x_2 & y_2 & \phi_2 & x_3 & y_3 & \phi_3 \end{bmatrix}^T$$

The points for the joints are described by:

$$s_1'^{\scriptscriptstyle \mathrm{pl}} = \begin{bmatrix} -\frac{L_1}{2} \\ 0 \end{bmatrix}, \quad s_1'^{\scriptscriptstyle \mathrm{p2}} = \begin{bmatrix} \frac{L_1}{2} \\ 0 \end{bmatrix}, \quad s_2'^{\scriptscriptstyle \mathrm{p2}} = \begin{bmatrix} -\frac{L_2}{2} \\ 0 \end{bmatrix}, \quad s_2'^{\scriptscriptstyle \mathrm{p3}} = \begin{bmatrix} \frac{L_2}{2} \\ 0 \end{bmatrix}, \quad s_3'^{\scriptscriptstyle \mathrm{p3}} = \begin{bmatrix} -\frac{L_3}{2} \\ 0 \end{bmatrix}, \quad s_3'^{\scriptscriptstyle \mathrm{p4}} = \begin{bmatrix} \frac{L_3}{2} \\ 0 \end{bmatrix},$$

so that e.g. $s_1^{p^2}$ describes the local point p2 in body 1.

The coordinates for the C-vectors are:

$$C_1 = \begin{bmatrix} 0 \\ 0 \end{bmatrix}, \quad C_2 = \begin{bmatrix} 0 \\ 0 \end{bmatrix}, \quad C_3 = \begin{bmatrix} 0 \\ 0 \end{bmatrix}, \quad C_4 = \begin{bmatrix} 20 \\ 0 \end{bmatrix}$$

And the coordinates for the r-vectors are:

$$m{r_1} = egin{bmatrix} x_1 \ y_1 \end{bmatrix}, \quad m{r_2} = egin{bmatrix} x_2 \ y_2 \end{bmatrix}, \quad m{r_3} = egin{bmatrix} x_3 \ y_3 \end{bmatrix}$$

The Kinematic constraint equations are as follows:

$$\Phi^{\text{abs1}}(q) = \begin{bmatrix} r_1 + A s_1'^{P^1} - C_1 \end{bmatrix} = \begin{bmatrix} x_1 - \frac{L_1}{2} \cos(\phi_1) \\ y_1 - \frac{L_1}{2} \sin(\phi_1) \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \end{bmatrix}$$

$$\Phi^{\text{rel1}}(q) = \begin{bmatrix} r_1 + A s_1'^{P^2} - \left(r_2 + A s_2'^{P^2} \right) - C_2 \end{bmatrix} = \begin{bmatrix} x_1 + \frac{L_1}{2} \cos(\phi_1) - \left(x_2 - \frac{L_2}{2} \cos(\phi_2) \right) \\ y_1 + \frac{L_1}{2} \sin(\phi_1) - \left(y_2 - \frac{L_2}{2} \sin(\phi_2) \right) \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \end{bmatrix}$$

$$\Phi^{\text{rel2}}(q) = \begin{bmatrix} r_2 + A s_2'^{P^3} - \left(r_3 + A s_3'^{P^3} \right) - C_3 \end{bmatrix} = \begin{bmatrix} x_2 + \frac{L_2}{2} \cos(\phi_2) - \left(x_3 - \frac{L_3}{2} \cos(\phi_3) \right) \\ y_2 + \frac{L_2}{2} \sin(\phi_2) - \left(y_3 - \frac{L_3}{2} \sin(\phi_3) \right) \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \end{bmatrix}$$

•
$$\Phi^{abs2}(q) = \begin{bmatrix} r_3 + As_3'^{P^4} - C_4 \end{bmatrix} = \begin{bmatrix} x_3 + \frac{L_3}{2}\cos(\phi_3) - 20 \\ y_3 + \frac{L_3}{2}\sin(\phi_3) \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \end{bmatrix}$$

As a system of constraints the vectorfunction the above equations become:

$$\Phi^K(q) = \begin{bmatrix} \Phi^{\mathrm{abs1}}(q) \\ \Phi^{\mathrm{rev1}}(q) \\ \Phi^{\mathrm{rev2}}(q) \\ \Phi^{\mathrm{abs2}}(q) \end{bmatrix} = \begin{bmatrix} x_1 - \frac{L_1}{2}\cos(\phi_1) \\ y_1 - \frac{L_1}{2}\sin(\phi_1) \\ x_1 + \frac{L_1}{2}\cos(\phi_1) - \left(x_2 - \frac{L_2}{2}\cos(\phi_2)\right) \\ y_1 + \frac{L_1}{2}\sin(\phi_1) - \left(y_2 - \frac{L_2}{2}\sin(\phi_2)\right) \\ x_2 + \frac{L_2}{2}\cos(\phi_2) - \left(x_3 - \frac{L_3}{2}\cos(\phi_3)\right) \\ y_2 + \frac{L_2}{2}\sin(\phi_2) - \left(y_3 - \frac{L_3}{2}\sin(\phi_3)\right) \\ x_3 + \frac{L_3}{2}\cos(\phi_3) - 20 \\ y_3 + \frac{L_3}{2}\sin(\phi_3) \end{bmatrix} = \begin{bmatrix} \vec{o} \\ 0 \\ 0 \\ 0 \\ 0 \\ 0 \end{bmatrix}$$

3. Setup of the driving constraint Φ^D that imposes $\phi_1=\omega t, \quad \omega=1.5 {{\rm rad} \over {\rm s}}$

As there is 1 DoF for the system, an absolute driving constraint is added to the system:

$$\Phi^D(q,t) = \left[\phi_1 \text{-}\omega t\right] = 0$$

4. Calculate the constraint jacobian Φ_q

First B is defined:

$$B(\phi) = \begin{bmatrix} -\sin(\phi) & -\cos(\phi) \\ \cos(\phi) & -\sin(\phi) \end{bmatrix}$$

by combining Φ^K and Φ^D into Φ and then taking the partial derivative with respect to q the constraint jacobian Φ_q can be obtained:

$$\begin{split} \boldsymbol{\Phi_q} &= \frac{\partial \Phi}{\partial q} = \begin{bmatrix} \frac{\partial \Phi^K}{\partial q} \\ \frac{\partial \Phi^D}{\partial q} \end{bmatrix} = \begin{bmatrix} I_{2x2} & B_1 s_1'^{\text{pl}} & 0 & 0 & 0 & 0 & 0 \\ I_{2x2} & B_1 s_1'^{\text{pl}} & -I_{2x2} & -B_2 s_2'^{\text{pl}} & 0 & 0 & 0 \\ 0 & 0 & I_{2x2} & B_2 s_2'^{\text{pl}} & -I_{2x2} & -B_3 s_3'^{\text{pl}} \\ 0 & 0 & 0 & 0 & I_{2x2} & B_3 s_3'^{\text{pl}} \\ 0 & 1 & 0 & 0 & 0 & 0 & 0 & 0 & 0 \end{bmatrix} \\ &= \begin{bmatrix} 1 & 0 & \frac{L_1}{2} \sin(\phi_1) & 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 1 & -\frac{L_1}{2} \cos(\phi_1) & 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 1 & -\frac{L_1}{2} \sin(\phi_1) & -1 & 0 & -\frac{L_2}{2} \sin(\phi_2) & 0 & 0 & 0 \\ 0 & 1 & \frac{L_1}{2} \cos(\phi_1) & 0 & -1 & \frac{L_2}{2} \cos(\phi_2) & 0 & 0 & 0 \\ 0 & 0 & 0 & 1 & 0 & -\frac{L_2}{2} \sin(\phi_2) & -1 & 0 & -\frac{L_3}{2} \sin(\phi_3) \\ 0 & 0 & 0 & 0 & 1 & \frac{L_2}{2} \cos(\phi_2) & 0 & -1 & \frac{L_3}{2} \cos(\phi_3) \\ 0 & 0 & 0 & 0 & 0 & 0 & 1 & 0 & -\frac{L_3}{2} \sin(\phi_3) \\ 0 & 0 & 0 & 0 & 0 & 0 & 0 & 1 & \frac{L_3}{2} \cos(\phi_3) \\ 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 1 & \frac{L_3}{2} \cos(\phi_3) \\ 0 & 0 & 1 & 0 & 0 & 0 & 0 & 0 & 0 & 0 \end{bmatrix} \end{split}$$

5. Setup of the velocity and acceleration equations ν and γ

First the equations for the velocities will be setup and then the accelerations.

Velocities:

The velocity equation is defined as:

$$u = -\Phi_t = \Phi_{\boldsymbol{q}}\dot{q}, \quad \text{where } \dot{q} = \begin{bmatrix} \dot{x_1} & \dot{y_1} & \dot{\phi_1} & \dot{x_2} & \dot{y_2} & \dot{\phi_2} & \dot{x_3} & \dot{y_3} & \dot{\phi_3} \end{bmatrix}^T$$

as only Φ^D is a function of time $(\phi_1 - \omega t)$ the vector $-\Phi_t$ becomes:

$$\boldsymbol{\nu} = \begin{bmatrix} \textbf{0} & w \end{bmatrix}^T = \boldsymbol{\Phi_q} \dot{\boldsymbol{q}}$$

By left multiplying with ${f \Phi_q}^{-1}$ the equation can be rewritten as to obtain $\dot q$:

$$\dot{q}=\Phi_q^{\;\;-1}
u$$

Which when written out in matrices is:

for which the velocities can be calculated for each time instance.

Accelerations:

The acceleration equations are defined as:

$$\boldsymbol{\gamma} = \boldsymbol{\Phi_q} \ddot{\boldsymbol{q}} = - \left(\boldsymbol{\Phi_q} \dot{\boldsymbol{q}}\right)_{\boldsymbol{q}} \dot{\boldsymbol{q}} - 2\boldsymbol{\Phi_{qt}} \dot{\boldsymbol{q}} - \boldsymbol{\Phi_{\text{tt}}}, \quad \text{ where } \ddot{\boldsymbol{q}} = \begin{bmatrix} \ddot{x_1} & \ddot{y_1} & \ddot{\phi_1} & \ddot{x_2} & \ddot{y_2} & \ddot{\phi_2} & \ddot{x_3} & \ddot{y_3} & \ddot{\phi_3} \end{bmatrix}^T$$

As $m{\Phi_q}$ does not contain t, $m{\Phi_{q\mathrm{t}}} = \left[ec{0}
ight]$ and as only $m{\Phi}^D$ is dependant on time:

$$\mathbf{\Phi}_{\mathrm{tt}} = \frac{\partial^2 \mathbf{\Phi}^D}{\partial^2 t} = \frac{\partial \mathbf{\Phi}_t^D}{\partial t} = \begin{bmatrix} \vec{0} \end{bmatrix}$$

The expression for γ then becomes: $\gamma=\Phi_q\ddot{q}=-ig(\Phi_q\dot{q}ig)_q\dot{q},$ where when solving for \ddot{q} :

$$\ddot{q}={f \Phi_q}^{-1}\Big(-igl({f \Phi_q}\dot{q}igr)_q\dot{q}\Big)$$

Before \ddot{q} can be solved for the term $-ig(\Phi_{\!m{q}}\dot{q}ig)_{m{q}}\dot{q}$ has to be calculated in the order:

1.
$$\Phi_q \dot{q}$$

2.
$$\left(\mathbf{\Phi}_{q}\dot{q}\right)_{q}$$

3.
$$-\left(\Phi_{q}\dot{q}\right)_{q}\dot{q}$$

First:

$$\boldsymbol{\Phi_{q}\dot{q}} = \begin{bmatrix} I_{2x2} & B_{1}s_{1}^{\prime p1} & 0 & 0 & 0 & 0 \\ I_{2x2} & B_{1}s_{1}^{\prime p2} & -I_{2x2} & -B_{2}s_{2}^{\prime p2} & 0 & 0 \\ 0 & 0 & I_{2x2} & B_{2}s_{2}^{\prime p3} & -I_{2x2} & -B_{3}s_{3}^{\prime p3} \\ 0 & 0 & 0 & 0 & I_{2x2} & B_{3}s_{3}^{\prime p3} \\ 0 & 1 & 0 & 0 & 0 & 0 \end{bmatrix} \begin{bmatrix} \boldsymbol{r_{1}} \\ \dot{\phi_{1}} \\ \boldsymbol{r_{2}} \\ \dot{\phi_{2}} \\ \boldsymbol{r_{3}} \\ \dot{\phi_{3}} \end{bmatrix} = \begin{bmatrix} \boldsymbol{x_{1}} + \dot{\phi_{1}} \frac{L_{1}}{2} \sin(\phi_{1}) \\ \boldsymbol{y_{1}} - \dot{\phi_{1}} \frac{L_{1}}{2} \cos(\phi_{1}) \\ \boldsymbol{x_{1}} - \dot{\phi_{1}} \frac{L_{1}}{2} \sin(\phi_{1}) - \boldsymbol{x_{2}} - \dot{\phi_{2}} \frac{L_{2}}{2} \sin(\phi_{2}) \\ \boldsymbol{y_{1}} + \dot{\phi_{1}} \frac{L_{1}}{2} \cos(\phi_{1}) - \boldsymbol{y_{2}} + \dot{\phi_{2}} \frac{L_{2}}{2} \cos(\phi_{2}) \\ \boldsymbol{x_{2}} - \dot{\phi_{2}} \frac{L_{2}}{2} \sin(\phi_{2}) - \boldsymbol{x_{3}} - \dot{\phi_{3}} \frac{L_{3}}{2} \sin(\phi_{3}) \\ \boldsymbol{y_{2}} + \dot{\phi_{2}} \frac{L_{2}}{2} \cos(\phi_{2}) - \boldsymbol{y_{3}} + \dot{\phi_{3}} \frac{L_{3}}{2} \cos(\phi_{3}) \\ \boldsymbol{x_{3}} - \dot{\phi_{3}} \frac{L_{3}}{2} \sin(\phi_{3}) \\ \boldsymbol{y_{3}} + \dot{\phi_{3}} \frac{L_{3}}{2} \cos(\phi_{3}) \\ \dot{\phi_{1}} \end{bmatrix}$$

Then

and lastly

$$\left(\Phi_{m{q}}\dot{m{q}}
ight)_{m{q}}\dot{m{q}}=$$

Then the expression for γ is:

$$\boldsymbol{\gamma} = - \left(\boldsymbol{\Phi}_{\boldsymbol{q}} \dot{\boldsymbol{q}} \right)_{\boldsymbol{q}} \dot{\boldsymbol{q}} = \begin{bmatrix} -\dot{\phi}_1 \frac{2L_1}{2} \cos(\phi_1) \\ -\dot{\phi}_1 \frac{2L_1}{2} \sin(\phi_1) \\ \dot{\phi}_1 \frac{2L_1}{2} \cos(\phi_1) - \dot{\phi}_2 \frac{2L_2}{2} \cos(\phi_2) \\ \dot{\phi}_1 \frac{2L_1}{2} \sin(\phi_1) - \dot{\phi}_2 \frac{2L_2}{2} \sin(\phi_2) \\ \dot{\phi}_2 \frac{2L_2}{2} \cos(\phi_2) - \dot{\phi}_3 \frac{2L_3}{2} \cos(\phi_3) \\ \dot{\phi}_2 \frac{2L_2}{2} \sin(\phi_2) - \dot{\phi}_3 \frac{2L_3}{2} \sin(\phi_3) \\ \dot{\phi}_3 \frac{2L_3}{2} \cos(\phi_3) \\ \dot{\phi}_3 \frac{2L_3}{2} \sin(\phi_3) \\ 0 \end{bmatrix}$$

With the accelerations solved for:

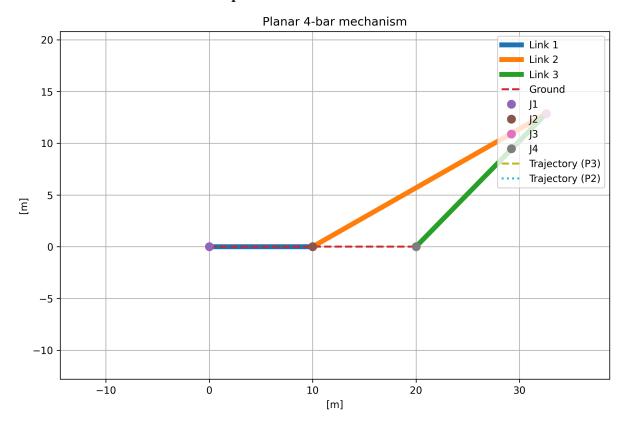
$$\ddot{\boldsymbol{q}} = \boldsymbol{\Phi_q}^{-1} \bigg(- \big(\boldsymbol{\Phi_q} \dot{\boldsymbol{q}}\big)_{\boldsymbol{q}} \dot{\boldsymbol{q}} \bigg)$$

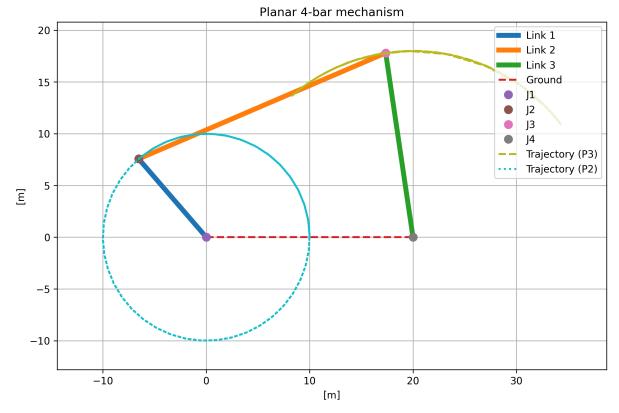
and with matrices:

$$\begin{bmatrix} \ddot{x_1} \\ \ddot{y_1} \\ \ddot{y_1} \\ \ddot{x_2} \\ \ddot{y_2} \\ \ddot{x_3} \\ \ddot{y_3} \\ \ddot{\phi_3} \end{bmatrix} = \begin{bmatrix} 1 & 0 & \frac{L_1}{2} \sin(\phi_1) & 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 1 & -\frac{L_1}{2} \cos(\phi_1) & 0 & 0 & 0 & 0 & 0 & 0 \\ 1 & 0 & -\frac{L_1}{2} \sin(\phi_1) & -1 & 0 & \frac{L_2}{2} \sin(\phi_2) & 0 & 0 & 0 & 0 \\ 0 & 1 & \frac{L_1}{2} \cos(\phi_1) & 0 & -1 & -\frac{L_2}{2} \cos(\phi_2) & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 1 & 0 & -\frac{L_2}{2} \sin(\phi_2) & -1 & 0 & \frac{L_3}{2} \sin(\phi_3) \\ 0 & 0 & 0 & 0 & 1 & \frac{L_2}{2} \cos(\phi_2) & 0 & -1 & -\frac{L_3}{2} \cos(\phi_3) \\ 0 & 0 & 0 & 0 & 0 & 0 & 1 & 0 & -\frac{L_3}{2} \sin(\phi_3) \\ 0 & 0 & 0 & 0 & 0 & 0 & 1 & 0 & -\frac{L_3}{2} \sin(\phi_3) \\ 0 & 0 & 0 & 0 & 0 & 0 & 1 & 0 & -\frac{L_3}{2} \sin(\phi_3) \\ 0 & 0 & 0 & 0 & 0 & 0 & 1 & \frac{L_3}{2} \cos(\phi_3) \\ 0 & 0 & 0 & 0 & 0 & 0 & 1 & \frac{L_3}{2} \cos(\phi_3) \\ 0 & 0 & 0 & 0 & 0 & 0 & 0 & 1 & \frac{L_3}{2} \cos(\phi_3) \\ 0 & 0 & 1 & 0 & 0 & 0 & 0 & 0 & 0 \end{bmatrix}$$

6. Code for the system of equations in python:

7. The motion of the mechanism plotted for 10 seconds:





8. Initial configuration of the mechanism and the trajectories of the revolute joints: