#### **MATLAB Script**

```
clear; close all;
%% O. Initialize Parameters
L = 1200; % Length of bridge
n = 1200; % Discretize into 1 mm seq.
P = 400; % Total weight of train [N]
x = linspace(0, L, n+1); % x-axis
%% 1. SFD, BMD under train loading
x train = [172 348 512 688 852 1028]; % Train Load Locations
P train = [1 \ 1 \ 1 \ 1 \ .35 \ 1.35] * P/6; % Change for force in each wheel,
first two are heaviest
n train = 3; % num of train locations
y SFD = zeros(1,1200);
% Solve for SFD and BMD with the train at different locations
moment a = 0;
for i = 1:length(x train)
   start_x = x_train(i);% start location of train, each wheel position
   moment_a = moment_a + start_x*P_train(i);% sum of moments at A eqn
end
%moment a final = moment b final, fby = moment a/1200
fby = moment a/1200;
```

```
% sum of Fy eqn
fay = sum(P_train) - fby;
```

```
P_train_neg = -1.*P_train;
%Total forces
load = [fay,P train neg(1), P train neg(2), P train neg(3),
P_train_neg(4), P_train_neg(5), P train neg(6), fby];
%Shear force values
SFD = zeros(8,1);
for i = 1:length(SFD)
  if i > 1
       SFD(i) = load(i) + SFD(i-1);
   else
       SFD(i) = load(i);
   end
end
%Force locations
x_{SFD} = [0, x_{train}(1), x_{train}(2), x_{train}(3), x_{train}(4), x_{train}(5),
x train(6), 1200];
```

```
for i = 1:1200
   if i == 1
        y_SFD(i) = 0;
   elseif i <=x_SFD(2)
        y_SFD(i) = SFD(1);
   elseif x_SFD(2)</pre>
   elseif x_SFD(2)
   value is a i<=x_SFD(3)
   yalue is a i<=x_SFD(4)
```

```
y SFD(i) = SFD(3);
   elseif x_SFD(4)<i & i<=x_SFD(5)</pre>
      y SFD(i) = SFD(4);
   elseif x_SFD(5)<i & i<=x_SFD(6)</pre>
      y SFD(i) = SFD(5);
   elseif x SFD(6)<i & i<=x SFD(7)</pre>
      y SFD(i) = SFD(6);
   elseif x_SFD(7)<i & i<x_SFD(8)</pre>
       y SFD(i) = SFD(7);
   elseif i==x_SFD(8)
      y_SFD(i) = SFD(8);
   end
end
x = 0:1:1199;
y = zeros(1, 1200);
figure
hold on
plot(x,y_SFD, "-r")
plot(x,y," r:")
hold off
% BMD
area = zeros(1,8);
for i = 1:8
  if i>1
      area(i) = SFD(i-1)*(x SFD(i)-x SFD(i-1)) + area(i-1);
   end
end
plot(x_SFD, area)
```

```
SFD_max = max(abs(SFD)); % SFD envelope
BMD = max(area); % BMD envelope
```

```
bottom = [75, 2*1.27]; %[width, height] in mm
area_bot = bottom(1)*bottom(2);
side = [1.27, 80];
area_side = side(1)*side(2);
tab_top = [15, 1.27];
area_tab_top = tab_top(1)*tab_top(2);
tab_bot = [15, 1.27];
area_tab_bot = tab_bot(1)*tab_bot(2);
top = [100, 2*1.27];
area_top = top(1)*top(2);
```

```
dimensions = [bottom, side, side, tab_top, tab_top, tab_bot, tab_bot,
top];
areas = [area_bot, area_side, area_side, area_tab_top, area_tab_top,
area_tab_bot, area_tab_bot, area_top];
```

```
%% 3. Calculate Sectional Properties
% ybar. location of centroidal axis from the bottom
h_tot = bottom(2) + side(2) + top(2);
%Centroidal axis of each subsection
y_bar_each = [bottom(2)/2, bottom(2)+side(2)/2, bottom(2)+side(2)/2,
bottom(2)+side(2) - tab_top(2)/2, bottom(2)+side(2) - tab_top(2)/2,
bottom(2)+tab_bot(2)/2, bottom(2)+tab_bot(2)/2, bottom(2)+side(2) +
top(2)/2];
```

```
ybar = (area_bot*(bottom(2)/2) + 2*area_side*(bottom(2)+side(2)/2) +
2*area_tab_top*(bottom(2)+side(2) - tab_top(2)/2) +
2*area_tab_bot*(bottom(2)+tab_bot(2)/2) + area_top*(bottom(2)+side(2) +
top(2)/2))/(area_bot+2*area_side+2*area_tab_top+2*area_tab_bot+area_top)
;%manually calculated
ybot = ybar;
ytop = (bottom(2)+side(2)+top(2)) - ybar;
```

```
% I value
I_each = zeros(1,8);
for i = 1:8
    I_each(i) = (dimensions(2*i-1)*dimensions(2*i)^3)/12;
end
```

```
d_each = zeros(1,8);
for i = 1:8
    d_each(i) = (ybar - y_bar_each(i));
end
% I = %sum of I + A*d^2
d_each_squared = d_each.^2;
I = sum(I_each);
for i = 1:8
    I = I + areas(i)*d_each_squared(i);
end
```

```
%Flexural stress = m*y/I
flex_stress_top = max(area)*ytop/I;
flex_stress_bot = max(area)*ybot/I;
```

```
%Shear stress
%centroid
Q = 0;
for i = 1:8
   Q = Q + areas(i)*abs(d each(i));
end
b = 1.27*2; %length of matter on centroidal axis ybar
shear centroid = SFD \max *Q/(I*b);
%glue tabs
b tab = 2*(5+1.27);
shear_glue = SFD_max*Q/(I*b_tab);
%% 5. Material and Thin Plate Buckling Capacities
E = 4000;
mu = 0.2;
S_tens = flex_stress_bot;
S_comp = flex_stress_top;
T_max = shear_centroid;
T_gmax = shear_glue;
%Buckling
%top flange (mid part) k=4
S_buck1 = ((4*pi^2*E)/(12*(1-mu^2)))*(top(2)/bottom(1))^2;
side_flange = (top(1)-bottom(1))/2; %length
S buck2 = ((0.425*pi^2*E)/(12*(1-mu^2)))*(top(2)/side flange)^2;
S buck3 = ((6*pi^2*E)/(12*(1-mu^2)))*(side(1)/ytop)^2;
```

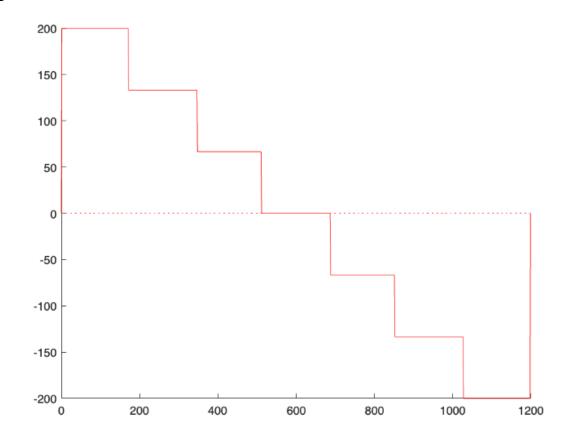
```
a = 210; %the distance between diaphragms (needs to be decided from design plan) T_{\text{buck}} = ((5*(pi^2)*E)/(12*(1-mu^2)))*((side(1)/a)+(side(1))/(side(2)+bottom(2))) ^2;
```

```
%% 6. FOS
FOS tens = 30/S tens
FOS comp = 6/S comp
FOS shear = 4/T max
FOS glue = 2/T gmax
FOS buck1 = S buck1/S tens
FOS buck2 = S_buck2/S_tens
FOS buck3 = S buck3/S tens
FOS_buckV = T_buck/T_max
% %% 7. Min FOS and the failure load Pfail
FOS_vector = [FOS_tens, FOS_comp, FOS_shear, FOS_glue, FOS_buck1,
FOS buck2, FOS buck3, FOS buckV];
minFOS = min(FOS vector)
%Failure Load
Ff= 4/(Q/(I*b));
percent = Ff/fby;
Pf = sum(P train)*percent
```

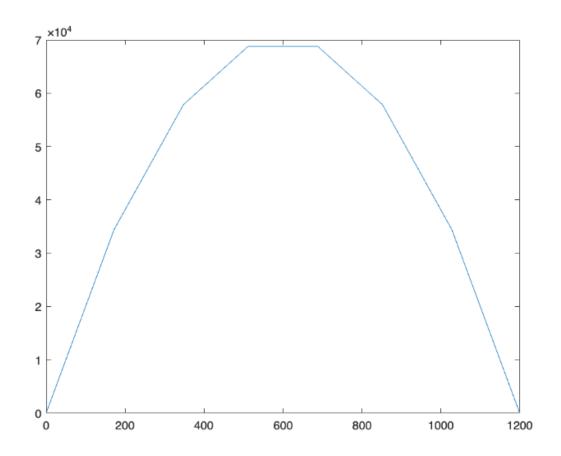
# MATLAB Output

# Design 0 under load case 1

SFD



**BMD** 



#### Intermediate Steps

fby = 200.0000 mm

fay = 200.0000 mm

 $SFD_max = 200.0000 N$ 

BMD = 6.8800e+04 N\*mm

ybar = 40.2935 mm

ybot = 40.2935 mm

ytop = 35.9765 mm

I = 8.4871e+04 mm<sup>4</sup>

S\_tens = 6.3346 MPa

S\_comp = 5.6560 MPa

 $T_{max} = 3.5417 \text{ MPa}$ 

 $T_gmax = 0.3587 MPa$ 

S buck1 = 3.4546 MPa

 $S_buck2 = 23.4911 MPa$ 

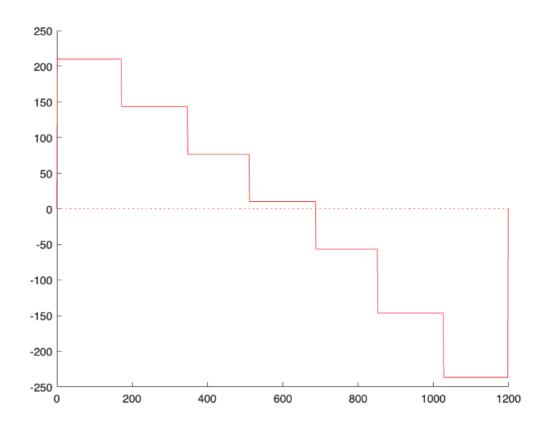
S\_buck3 = 25.6229 MPa

T buck = 9.0493 MPa

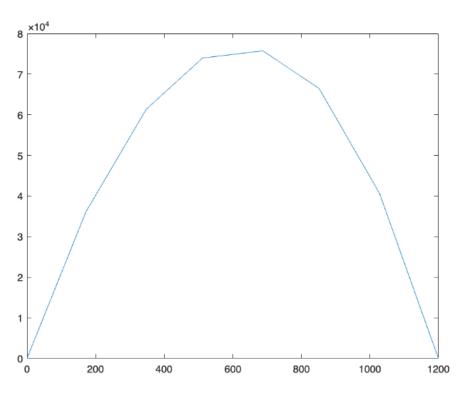
FOS\_tens = 4.7359 FOS\_comp = 1.0608 FOS\_shear = 1.1294 FOS\_glue = 5.5758 FOS\_buck1 = 0.5453 FOS\_buck2 = 3.7084 FOS\_buck3 = 4.0449 FOS\_buckV = 2.5550 minFOS = 0.5453

### Final design under load case 2

SFD



BMD



FOS\_tens = 5.7118

FOS\_comp = 1.8475

FOS\_shear = 1.8003

FOS\_glue = 4.4441

FOS\_buck1 = 2.9934

FOS\_buck2 = 11.4497

FOS\_buck3 = 6.1577

 $FOS_buckV = 3.6230$ 

minFOS = 1.8003

Pf = 804.1352N