

A Continuum Manipulator with Closed-form Inverse Kinematics and Independently Tunable Stiffness

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Abstract— Continuum manipulators can accomplish various tasks in confined spaces, benefiting from their compliant structures and improved dexterity. Confined and unstructured spaces may require both enhanced stiffness of a continuum manipulator for precision and payload, as well as compliance for safe interaction. Thus, studies have been consistently dedicated to design continuum or articulated manipulators with tunable stiffness to adapt to different operating conditions. This paper presents a continuum manipulator with independently tunable stiffness where the stiffness variation does not affect the movement of the manipulator's end-effector. Moreover, the proposed continuum manipulator is found to have analytical inverse kinematics. The design concept, analytical kinematics, system construction and experimental characterizations are presented. The results showed that the manipulator's stiffness can be increased up to 3.61 times of the minimal value, demonstrating the effectiveness of the proposed idea.

I. INTRODUCTION

Continuum manipulators have been of interest due to their improved dexterity in confined spaces and intrinsic compliant interaction in unstructured spaces. For example, slender continuum manipulators have been applied in surgical robots [1] and field operations [2, 3]. The continuum manipulator with tunable stiffness is desired to adapt to different operating conditions. Enhanced stiffness of a continuum manipulator is desired for precision and payload, while the compliance is preferred for safe interaction, preventing possible damages to fragile nearby objects or structures.

Hence, recent researches have focused on exploring possible designs for manipulators with tunable stiffness. For example, active materials can be used as tunable stiffness elements, including magnetorheological fluids [4], electrorheological fluids [5] or thermally softened alloy or plastics [6, 7]. However, the uses of active materials usually complicate the robotic system. In addition, the response can be slow for thermally activated materials (on the order of seconds).

Tunable stiffness can also be achieved by controlling the friction between the structural elements. For example, in

tendon-driven manipulators, high tension forces from the pulling wires compress the vertebrae such that higher friction is generated from the contacting surfaces [8-12]. On the other hand, jamming can be used to introduce higher friction via pressurization for tunable stiffness [13-16]. However, the friction-based stiffness enhancement often leads to actuation hysteresis.

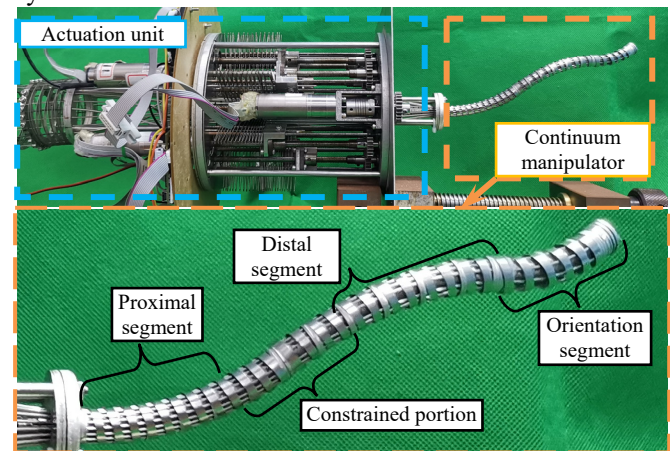


Fig. 1. The constructed continuum manipulator with closed-form inverse kinematics and independently tunable stiffness

It is possible to achieve tunable stiffness via proper controller design. For example, a modified position controller drove a concentric-tube manipulator into different poses for different stiffness upon understanding its mechanics [17]. A stiffness controller utilized the manipulator's intrinsic force sensing capability that was proposed in [18] to realize variable stiffness [19]. The controller-based approaches can be challenging in formulating the stiffness model based on the manipulator's mechanics.

Furthermore, stiffness can be enhanced via structure variations, such as the inclusion of a motion constraining kinematic chain [20]. However, such approach often complicated the system construction.

This paper hence proposes a continuum manipulator design with independently tunable stiffness. The manipulator with its actuation unit is shown in Fig. 1. As detailed in Section II, the manipulator's stiffness can be continuously tuned by moving a rigid tube inside the manipulator such that the constrained portion can be at different positions along the manipulator. It was found that the changed positions of the constrained portion will not change the position and orientation of the orientation segment, even though the manipulator's shape is indeed changed. This is hence referred to as independently

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visualization clarity, only three backbones for each segment are shown in the Fig. 2 and Fig. 3. The number of backbones in the prototype is different.

TABLE I
NOMENCLATURE USED IN THE KINEMATICS MODEL

| Symbol | Definition |
|----------------------|---|
| i | Index of the continuum segments. $i = p, d$ and o . The index p means the proximal segment, while d stands for the distal segment and o stands for the orientation segment. |
| l_i | l_p, l_d and l_o refer to the lengths of the PS, DS, and orientation segment, measured along the virtual central backbone. |
| l_r | The length of the constrained portion. |
| l_{total} | $l_{total} = l_p + l_d + l_r + l_o$ |
| θ_i | Bending angle of the continuum segment |
| δ_i | Rotation angle from $\hat{\mathbf{y}}_{i1}$ to $\hat{\mathbf{x}}_{ib}$ along $\hat{\mathbf{z}}_{ib}$. |
| Ψ_p | Configuration vectors of the IDCM: $\Psi_p = [\theta_p \delta_p]^T$ |
| Ψ_o | Configuration vector of the orientation segment. $\Psi_o = [\theta_o \delta_o]^T$ |
| Ψ | Configuration vector of continuum manipulator. $\Psi = [\Psi_p^T \Psi_o^T]^T$ |
| c_θ, s_θ | $\cos(\theta), \sin(\theta)$ |

- Base Disk Coordinate $\{ib\} \equiv \{\hat{\mathbf{x}}_{ib}, \hat{\mathbf{y}}_{ib}, \hat{\mathbf{z}}_{ib}\}$ is attached to the segment's base disk with its origin at the disk's center. $\hat{\mathbf{x}}_{ib}$ points from the center to the first backbone.
- Bending Plane Coordinate-1 $\{i1\} \equiv \{\hat{\mathbf{x}}_{i1}, \hat{\mathbf{y}}_{i1}, \hat{\mathbf{z}}_{i1}\}$ shares origin with $\{ib\}$ and has the continuum segment bent in its XY plane.

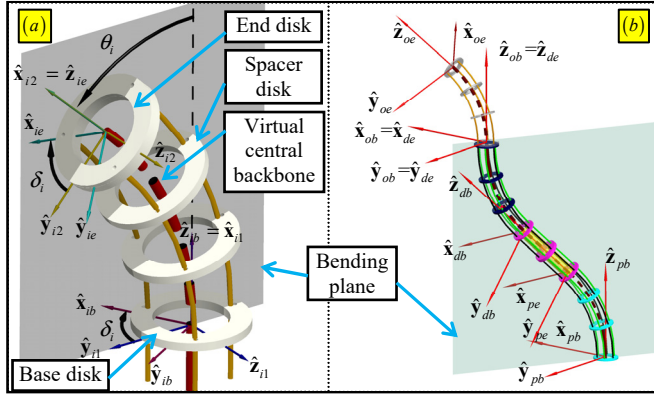


Fig. 3. Coordinates attachment and nomenclature of (a) the single continuum segment, and (b) the continuum manipulator.

- Bending Plane Coordinate-2 $\{i2\} \equiv \{\hat{\mathbf{x}}_{i2}, \hat{\mathbf{y}}_{i2}, \hat{\mathbf{z}}_{i2}\}$ is obtained from $\{i1\}$ by a rotation about $\hat{\mathbf{z}}_{i1}$ such that $\hat{\mathbf{x}}_{i2}$ becomes the virtual central backbone tangent at the end disk.
- End Disk Coordinate $\{ie\} \equiv \{\hat{\mathbf{x}}_{ie}, \hat{\mathbf{y}}_{ie}, \hat{\mathbf{z}}_{ie}\}$ is attached to the end disk. $\hat{\mathbf{x}}_{ie}$ points from the center to the first backbone, and $\hat{\mathbf{z}}_{ie}$ is normal to the end disk.

Two modeling assumptions are used.

- A virtual central backbone characterizes the length and shape of the continuum segment. The kinematics assumes a circular shape for the segments, referring to [32].
- The position of the constrained portion along the virtual central backbone is determined by the length of the PS, l_p . The lengths of the PS and the DS can be continuously changed via the insertion of the rigid tube.

B. Kinematics of a single continuum segment

The continuum segments, including the DS, the PS and the orientation segment, are inextensible with two bending DoFs, specified by the configuration vector $\Psi_i \equiv [\theta_i \delta_i]^T$. The length is l_i . The position of $\{ie\}$ in $\{ib\}$, ${}^{ib}\mathbf{p}_{ie}$, is written as in (1).

$${}^{ib}\mathbf{p}_{ie} = (l_i/\theta_i) \begin{bmatrix} c_{\delta_i}(1-c_{\theta_i}) & s_{\delta_i}(c_{\theta_i}-1) & s_{\theta_i} \end{bmatrix}^T \quad (1)$$

Where ${}^{ib}\mathbf{p}_{ie} = [0 \ 0 \ l_i]^T$, when $\theta_i = 0$.

The orientation of $\{ie\}$ in $\{ib\}$ is written as in (2), referring to the previous study [26].

$${}^{ib}\mathbf{R}_{ie} = \begin{bmatrix} c_{\theta_i}(c_{\delta_i})^2 + (s_{\delta_i})^2 & s_{\delta_i}c_{\delta_i}(1-c_{\theta_i}) & c_{\delta_i}s_{\theta_i} \\ s_{\delta_i}c_{\delta_i}(1-c_{\theta_i}) & c_{\theta_i}(s_{\delta_i})^2 + (c_{\delta_i})^2 & -s_{\delta_i}s_{\theta_i} \\ -c_{\delta_i}s_{\theta_i} & s_{\delta_i}s_{\theta_i} & c_{\theta_i} \end{bmatrix} \quad (2)$$

The inverse kinematics of a single continuum segment can be derived as follows to obtain the configuration vector Ψ_i from a given orientation ${}^{ib}\mathbf{R}_{ie}$.

$$\theta_i = \arccos({}^{ib}\mathbf{R}_{ie,33}) \quad (3)$$

$$\delta_i = \text{atan2}(-{}^{ib}\mathbf{R}_{ie,23}, {}^{ib}\mathbf{R}_{ie,13})$$

Where the function atan2 gives the angle between the positive x-axis and the ray passing the point $(-{}^{ib}\mathbf{R}_{ie,23}, {}^{ib}\mathbf{R}_{ie,13}) \neq (0,0)$. It should be noted that the use of the function atan2 can eliminate the algorithmic singularity introduced by an arctan function.

C. Kinematics of the IDCM

Translational movements of the IDCM would be generated due to the identical bending of the DS and the PS. Hence, configuration of the IDCM is specified by the configuration vector of the PS $\Psi_p \equiv [\theta_p \delta_p]^T$. The IDCM's shape can be described by a path in the bending plane sequentially along:

- The virtual central backbone of the PS (a circular arc of length l_p and bending angle θ_p), and the position ${}^{pb}\mathbf{p}_{pe}$ and orientation ${}^{pb}\mathbf{R}_{pe}$ can be calculated referring to (1) and (2);
- The axis of constrained portion with a distance of l_r , hence, the $\{db\}$ is located in $\{pe\}$ at ${}^{pe}\mathbf{p}_{db} = [0 \ 0 \ l_r]^T$; and
- The virtual central backbone of the DS (a circular arc of length l_d and bending angle $\theta_d = \theta_p$), and the tip position ${}^{db}\mathbf{p}_{de}$ also can be obtained according to (1).

As the bending plane is represented by δ_p , the expression of ${}^{pb}\mathbf{p}_{de}$ is hence derived as in (4).

$$\begin{aligned} {}^{pb}\mathbf{p}_{de} &= {}^{pb}\mathbf{p}_{pe} + {}^{pb}\mathbf{R}_{pe}({}^{pe}\mathbf{p}_{db} + {}^{db}\mathbf{p}_{de}) \\ &= \begin{bmatrix} c_{\delta_p}((1-c_{\theta_p})(l_p+l_d)/\theta_p + l_r s_{\theta_p}) \\ -s_{\delta_p}((1-c_{\theta_p})(l_p+l_d)/\theta_p + l_r s_{\theta_p}) \\ s_{\theta_p}(l_p+l_d)/\theta_p + l_r c_{\theta_p} \end{bmatrix} \end{aligned} \quad (4)$$

Where ${}^{pb}\mathbf{p}_{de} = [0 \ 0 \ l_p+l_d+l_r]^T$, when $\theta_p = 0$.

Let $l_b = l_p+l_d$ is the total length of the PS and DS. The l_b is constant in the design, regardless of the changes of l_p .

Since the tip movement of the IDCM is purely translational, the orientation of $\{de\}$ w.r.t $\{pb\}$, ${}^{pb}\mathbf{R}_{de}$, is an identity matrix.

$${}^{pb}\mathbf{R}_{de} = \mathbf{I}_{3 \times 3} \quad (5)$$

It's worth noting that the location of the rigid tube, l_p , does not affect the position ${}^{db}\mathbf{p}_{de}$ and the orientation ${}^{pb}\mathbf{R}_{de}$. Referring to (4), l_p alone does not change the value of ${}^{db}\mathbf{p}_{de}$. As long as $l_b = l_p + l_d$ is constant, altering the position of the constrained portion could be used to achieve the stiffness variation without affecting the movements of the continuum manipulator.

The closed-form inverse kinematics of the IDCM can be derived as follows.

The orientation of the end effector of the continuum manipulator is determined by the orientation segment, since the IDCM only provides translations. Then, when the desired pose of the continuum manipulator is given, the configuration of the orientation segment can be obtained from the desired orientation. Next, tip position of the IDCM can be calculated from the determined shape of the orientation segment.

Observing (4), it can be seen that δ_p , from the configuration vector $\boldsymbol{\Psi}_p \equiv [\theta_p \ \delta_p]^T$, can be easily obtained as in (6).

$$\delta_p = \text{atan} 2(-{}^{pb}\mathbf{p}_{de}|_y, {}^{pb}\mathbf{p}_{de}|_x) \quad (6)$$

Then, δ_p can be eliminated as in (7).

$$r_s = +\sqrt{{}^{pb}\mathbf{p}_{de|x}^2 + {}^{pb}\mathbf{p}_{de|y}^2} = (1 - c_{\theta_p})l_b/\theta_p + l_r s_{\theta_p} \quad (7)$$

It was found that the θ_p term can be further eliminated as in (8). Next, the half angle formulas can be used to generate (10).

$$\frac{r_s - l_r s_{\theta_p}}{{}^{pb}\mathbf{p}_{de}|_z - l_r c_{\theta_p}} = \frac{(1 - c_{\theta_p})\frac{l_b}{\theta_p}}{s_{\theta_p}\frac{l_b}{\theta_p}} = \frac{(1 - c_{\theta_p})}{s_{\theta_p}} \quad (8)$$

Equation (8) leads to (9).

$$\begin{aligned} (r_s - l_r s_{\theta_p})s_{\theta_p} &= ({}^{pb}\mathbf{p}_{de}|_z - l_r c_{\theta_p})(1 - c_{\theta_p}) \\ \Rightarrow r_s s_{\theta_p} - l_r s_{\theta_p}^2 &= {}^{pb}\mathbf{p}_{de}|_z - l_r c_{\theta_p} - c_{\theta_p} {}^{pb}\mathbf{p}_{de}|_z + l_r c_{\theta_p}^2 \\ \Rightarrow r_s s_{\theta_p} &= ({}^{pb}\mathbf{p}_{de}|_z + l_r)(1 - c_{\theta_p}) \end{aligned} \quad (9)$$

Then, Equation (10) can be obtained by substituting the half angle formulas of $\sin\theta_p = 2\sin(\theta_p/2)\cos(\theta_p/2)$ and $\cos\theta_p = 1 - 2\sin^2(\theta_p/2)$. θ_p is hence obtained as in (11).

$$\tan(\theta_p/2) = r_s / ({}^{pb}\mathbf{p}_{de}|_z + l_r) \quad (10)$$

$$\theta_p = 2 \cdot \text{atan} 2(r_s, {}^{pb}\mathbf{p}_{de}|_z + l_r) \quad (11)$$

D. Kinematics of the continuum manipulator

The orientation segment is stacked on top of the IDCM to form the proposed continuum manipulator. $\{ob\}$ coincides with $\{de\}$. Hence, the tip position ${}^{pb}\mathbf{p}_{oe}$ of the continuum manipulator can be calculated as follows.

$${}^{pb}\mathbf{p}_{oe} = {}^{pb}\mathbf{p}_{de} + {}^{pb}\mathbf{R}_{de} {}^{ob}\mathbf{p}_{oe} = {}^{pb}\mathbf{p}_{de} + {}^{ob}\mathbf{p}_{oe} \quad (12)$$

The orientation ${}^{pb}\mathbf{R}_{oe}$ of $\{oe\}$ in $\{pb\}$ is obtained as follows.

$${}^{pb}\mathbf{R}_{oe} = {}^{pb}\mathbf{R}_{de} {}^{ob}\mathbf{R}_{oe} = {}^{ob}\mathbf{R}_{oe} \quad (13)$$

The closed-form inverse kinematics of the proposed continuum manipulator is derived as follows.

Firstly, the configuration vector of the orientation segment, $\boldsymbol{\Psi}_o$, is obtained as in (3), since the orientation of the proposed continuum manipulator only depend on the orientation

segment. Then, tip position of the IDCM, ${}^{pb}\mathbf{p}_{de}$, is calculated from ${}^{pb}\mathbf{p}_{oe}$ and $\boldsymbol{\Psi}_o$ as in (14).

$${}^{pb}\mathbf{p}_{de} = {}^{pb}\mathbf{p}_{oe} - {}^{ob}\mathbf{p}_{oe} \quad (14)$$

Finally, the configuration vector $\boldsymbol{\Psi}_p$ of the IDCM can be calculated according to (6) and (11) with ${}^{pb}\mathbf{p}_{de}$ obtained from (14). The closed-form inverse kinematics is fully obtained.

The parameters of the manipulator are listed in TABLE II.

TABLE II
PARAMETERS OF THE CONTINUUM MANIPULATOR

| | | | |
|------------------------------------|--|------------------------------------|--|
| $l_b = 110 \text{ mm}$ | $l_r = 50 \text{ mm}$ | $l_o = 35 \text{ mm}$ | $l_p \in [30 \text{ mm}, 80 \text{ mm}]$ |
| $\theta_p \in [0^\circ, 90^\circ]$ | $\delta_p \in (-180^\circ, 180^\circ]$ | $\theta_o \in [0^\circ, 90^\circ]$ | $\delta_o \in (-180^\circ, 180^\circ]$ |

IV. DESIGN DESCRIPTION

The continuum manipulator shown in Fig. 4 with closed-form inverse kinematics and independently tunable stiffness is driven by an actuation unit shown in Fig. 5. The actuation unit alters the position of the constrained portion as well. The control infrastructure is also briefly introduced.

A. Continuum Manipulator

The schematic of the continuum manipulator which is formed by serially connecting the IDCM and the orientation segment is shown in Fig. 4(b).

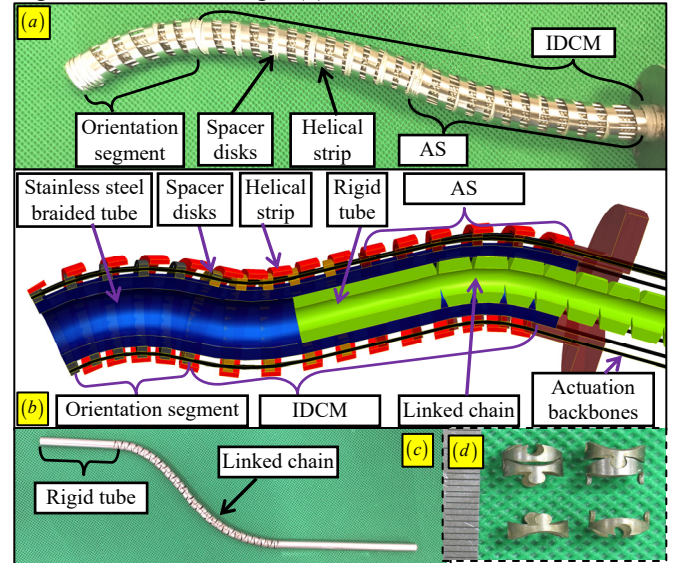


Fig. 4. Design details: (a) the continuum manipulator without the constrained portion, (b) the schematic, (c) the linked chain, and (d) the links

In this specific IDCM design, the AS from Section II is merged with the PS. The IDCM possesses four structural nitinol backbones, and its AS is bent by actuating four actuation backbones. These backbones all have diameter of 0.7mm. The orientation segment has eight structural backbones. As shown in Fig. 4(a), a stainless steel helical strip with the wall thickness of 0.1 mm was utilized to separate the spacer disks. A stainless steel braided tube is attached inside the spacer disks. It is flexible enough to be bent and it provides a smooth and continuous surface for inserting the stainless steel rigid tube to constrain the PS and DS. The rigid tube has an outer diameter of 7mm and an inner diameter of 6 mm.

An important enabling mechanism is to allow the translation of the rigid tube inside the IDCM without affecting bending the PS of the IDCM. Here a linked chain was designed as shown in Fig. 4(c). The linked chain is composed of articulated links that are cut from a tube, as shown in Fig. 4(d). The linked chain has high axial rigidity for transmitting pushing and pulling, while it has near-zero bending stiffness such that bending of the continuum segment is not affected.

The linked chain is connected with the stainless steel rigid tube so that the actuation unit translates the linked chain to push or pull the rigid tube as shown in Fig. 5.

B. Actuation Unit

The actuation unit in Fig. 5 is designed to actuate the continuum manipulator as well as alter the position of the constrained portion to vary the stiffness.

An Orientation Proximal Segment (OPS) is integrated to bend the orientation segment. The OPS and the orientation segment formed a dual continuum mechanism, as proposed in [30]. In the design, the OPS is bent by pushing or pulling four actuation backbones simultaneously. The backbones are arranged 90° apart and are attached to the end disk of the OPS.

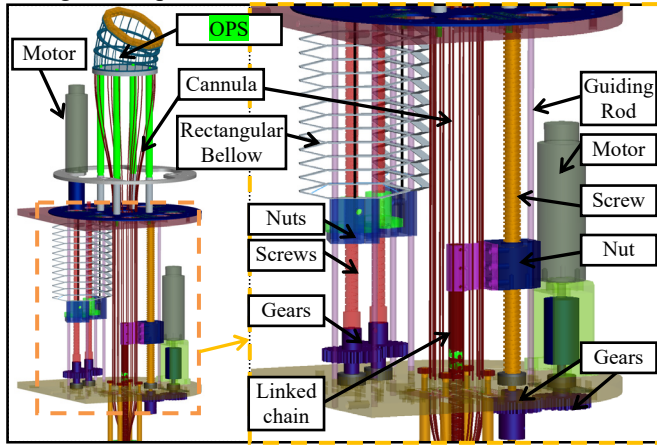


Fig. 5. Schematic of the actuation unit

To push and pull the actuation backbones in the OPS and the AS of the IDCM, four pairs of lead screws were used (two pairs for driving the IDCM's AS and two pairs for driving the OPS). Each pair of the lead screws is coupled via a meshing pair of spur gears, in order to make the nuts simultaneously and equally move in the opposite directions. The actuation backbones are fixed to the nuts, routed through rectangular bellow and a few cannulae. The rectangular bellow can slide on the guiding rods and prevent buckling of the backbones.

The rigid tube is pushed and pulled by the linked chain that is actuated by a motorized lead screw, as shown in Fig. 5.

Five Maxon DCX22L motors were integrated to actuate the screws (two for driving OPS, two for driving the IDCM's AS, and one for changing the position of the constrained portion), while five Maxon EPOS2 24/2 digital controllers were used to drive and control the motors. The desired positions of the motors are calculated in a desktop computer according to the kinematics and transmitted to the digital controllers via a CAN (Controller Area Network) bus.

V. EXPERIMENTAL CHARACTERIZATIONS

In order to verify the proposed design concept, numerical simulations and experimental validations were carried out. The simulation of the inverse kinematics is presented in Section V.A. The verification of independently tunable stiffness is presented in Section V.B.

A. Simulation of the Inverse Kinematics

The inverse kinematics of the continuum manipulator in different cases is simulated to show its motions.

In the first case, the end disk of the continuum manipulator is commanded to move on a given plane defined as in (15) and (16), as shown in Fig. 6. It's desired that its end disk of the orientation segment coincides with the given plane.

$$({}^{pb}\mathbf{p}_{oe} - \mathbf{p}_0) \times \mathbf{n} = 0 \quad (15)$$

$${}^{pb}\mathbf{R}_{oe}\mathbf{e}_3 = \mathbf{n} \quad (16)$$

Where $\mathbf{e}_3 = [0 \ 0 \ 1]^T$ and $\mathbf{n} = [0.577 \ 0.577 \ 0.577]^T$ is a normal vector that is perpendicular to the desired plane, and $\mathbf{p}_0 = [50.6 \text{ mm} \ 50.6 \text{ mm} \ 50.6 \text{ mm}]^T$ is a point located in the plane.

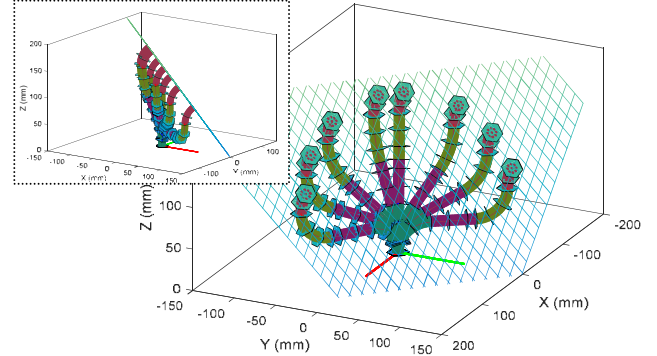


Fig. 6. Simulation of the continuum manipulator configurations corresponding to the manipulator's movements on the given plane

Since the normal vector \mathbf{n} is given, the configuration vector $\boldsymbol{\psi}_o$ of the orientation segment can be obtained according to (3). Then the feasible $\boldsymbol{\psi}_p$ values were obtained via iterating θ_p and solving the corresponding δ_p to check whether (15) holds.

The second case is a motion planning in the configuration space. When the target pose (${}^{pb}\mathbf{p}_{oe}^{tgt}$ and ${}^{pb}\mathbf{R}_{oe}^{tgt}$) is given, its corresponding configuration vector $\boldsymbol{\psi}^{tgt}$ can be calculated according to the closed-form inverse kinematics in Section III.

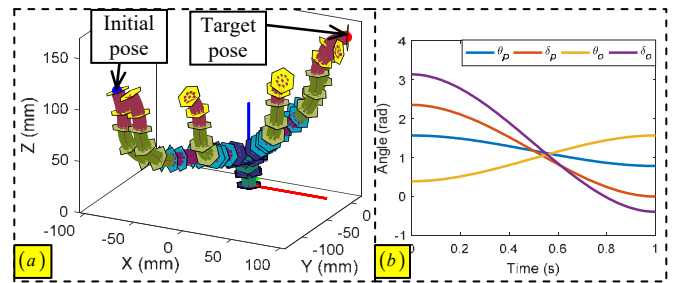


Fig. 7. Simulation of the motion planning in the configuration space: (a) intermediate poses, (b) trajectories of the configuration vector $\boldsymbol{\psi}$

The planning algorithm acts on the configuration vector and satisfies the following boundary conditions:

$$\psi(t_0)=\psi^{cur}; \dot{\psi}(t_0)=0; \psi(t_f)=\psi^{tgt}; \dot{\psi}(t_f)=0 \quad (17)$$

Where $t_0 = 0$ s is the start time, while $t_f = 1$ s is the end time.

Then a third order polynomial is applied to generate the trajectories of the configuration vector according to (18).

$$\begin{aligned} \psi(t) &= a_3 t^3 + a_2 t^2 + a_1 t + a_0 \\ \dot{\psi}(t) &= 3a_3 t^2 + 2a_2 t + a_1 \end{aligned} \quad (18)$$

Simulation of the continuum manipulator configurations with intervals of 0.2 s is plotted in Fig. 7(a). And the trajectories of the configuration vector are shown in Fig. 7(b).

B. Verification of the Independently Tunable Stiffness

Stiffness of the continuum manipulator was quantified under different configurations to demonstrate the effectiveness of the proposed stiffness variation approach.

Firstly, the continuum manipulator was driven to a desired configuration. Then, the position of the constrained portion l_p would be changed to alter the stiffness, as shown in Fig. 8(a). The orientation segment would be kept straight in the study.

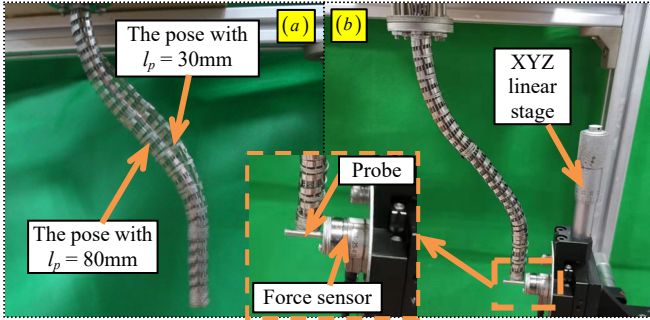


Fig. 8. Verification of tunable stiffness: a) overlapped image of the continuum robots with different positions of constrained portion, and b) experimental setup of stiffness quantification

A 6-axis force sensor (Nano-17, ATI Industrial Automation) with a probe was utilized to measure the exerted force. As shown in Fig. 8(b), the probe on the force sensor was driven by the XYZ linear stage to touch the continuum manipulator's tip, and then move its tip in the X, Y and Z directions in the $\{pb\}$ respectively. For every 0.5 mm perturbation, the exerted forces were measured and recorded, and the total perturbation was 3 mm. The slope between the measured forces and the given movements can be fitted to estimate the stiffness.

The tunable stiffness was verified under three configurations as shown in Fig. 9(a). The stiffness results with respect to the different positions of the constrained portion are plotted in Fig. 9(b), ranging from 30 mm to 80 mm with intervals of 10 mm. A few observations can be made.

- As shown in Fig. 8(a), changing the position of the constrained portion would slightly perturb the orientation segment. The perturbation may results from the friction between the disks and backbones, and the spacing between the rigid tube and the stainless steel braided tube.
- It can be seen that the continuum manipulator with a shorter proximal segment possesses higher stiffness.
- Changing the position of the constrained portion of the IDCM can effectively adjust the tip stiffness of continuum manipulator. The stiffness is increased from 1.34 times in

the X-direction of Config-2 to 3.61 times in the Y-direction of Config-3 of the minimal values.

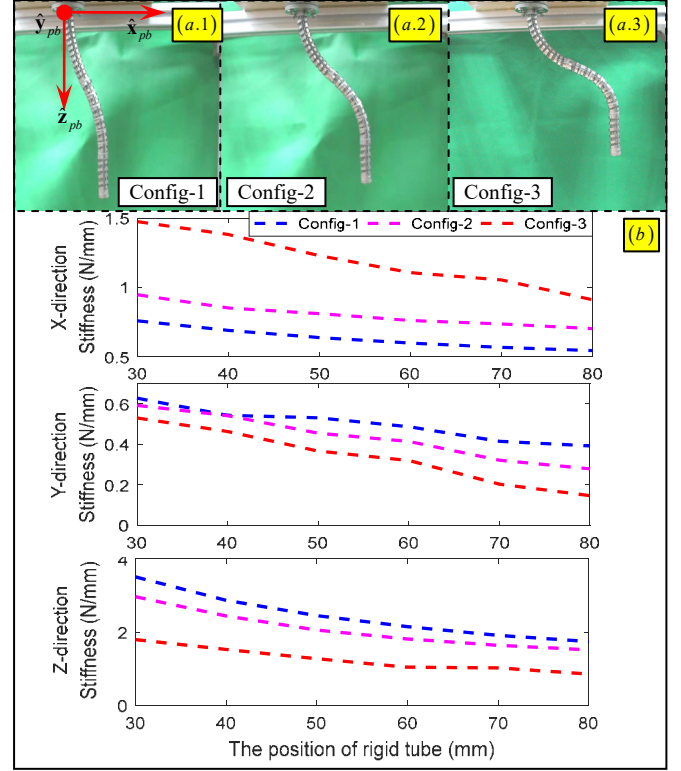


Fig. 9. Stiffness characterizations: a) poses under three configurations: (a.1) Config-1, (a.2) Config-2, and (a.3) Config-3; b) the stiffness results in the X, Y and Z directions with respect to different l_p lengths (a.k.a, positions of the constrained portion) from 30mm to 80mm

VI. CONCLUSION

Continuum manipulators have become popular in various applications in confined spaces because of their inherently safe interactions and distal dexterity. They are at times expected to have tunable stiffness to handle different tasks. For example, in surgical applications, high stiffness can be used for accurate tissue dissection and suture penetration, while low stiffness can be used for tissue separation to prevent damaging the tissue.

The paper proposes a continuum manipulator with closed-form inverse kinematics and independently tunable stiffness. An inverted dual continuum mechanism is proposed with a constrained portion. The independently tunable stiffness is achieved by altering the position of the constrained portion without affecting the tip pose of the continuum manipulator. The design concept, system construction, closed-form kinematics and experimental validations are detailed. Experimental results show that the continuum manipulator's tip stiffness could be varied up to 3.61 times of the minimal value, demonstrating the effectiveness of the proposed stiffness variation approach.

Future works mainly include two aspects. First, a detailed mechanics modeling would be derived to reflect the stiffness accurately to achieve stiffness control. Second, the proposed idea would be applied to a practical design of a continuum surgical manipulator to further verify the concept's usefulness.

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