



# *2A - Automatique*

## Chapter 2

# Control Science (AUT)

## Frequency-domain approach

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# Preamble

## About this course



## Course outline

- Effect of loop closing
- Concept of sensitivities
- Stability and the Nyquist Criterion



# Preamble

## Introduction example 1

We want to control the position of a motor to a setpoint  $y^c$ , with a position sensor. The model is as follows :

$$y = \frac{G_0}{p(1 + \tau_1 p)(1 + \tau_2 p)}(u - w)$$

- $u$  : supply voltage,  $w$  : disturbance input,  $y$  : angular position
- $G_0$  : speed gain,  $\tau_1$  electrical time constant,  $\tau_2$  mechanical time constant

### What can we tell about this system ?

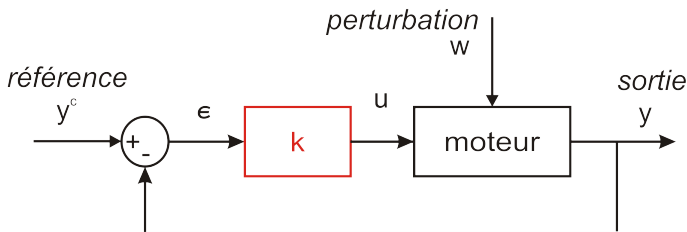
- Unstable (BIBO) in open loop : there is an integrator
- The slightest disturbance causes the motor to deviate infinitely far from its setpoint



# Preamble

## Introduction example 1

- We then propose to control the motor, by a control  $u = k(y^c - y)$ , with  $k$  a gain to adjust



## What does intuition tell us ?

- The higher the  $k$ , the better the performance will be
- But the higher the  $k$ , the greater the control effort will be

- And why not a small simulation study? *Matlab is coming . . .*

### The code

#### The model :

- `go=5 ; tau1=0.03 ; tau2=0.005 ;`
- `p=tf('p') ;`
- `sys=go/(p*(1+tau1*p)*(1+tau2*p)) ;`

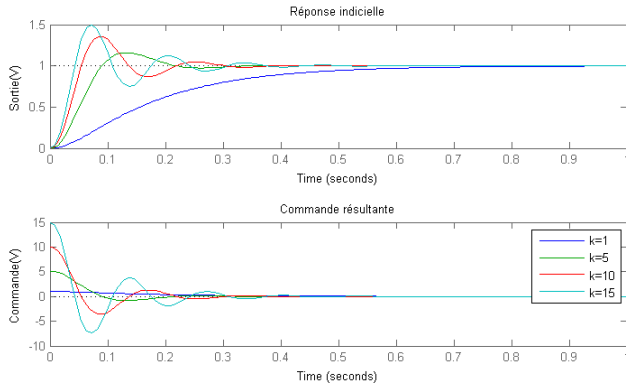
#### Feedback - relationship between output and setpoint :

- `k=1 ;`
- `sysbf=feedback(k*sys,1) ;`
- `step(sysbf,1)`

#### Feedback - relationship between control input and setpoint :

- `k=1 ;`
- `sysbfcom=feedback(k,sys) ;`
- `step(sysbfcom,1)`





## Question

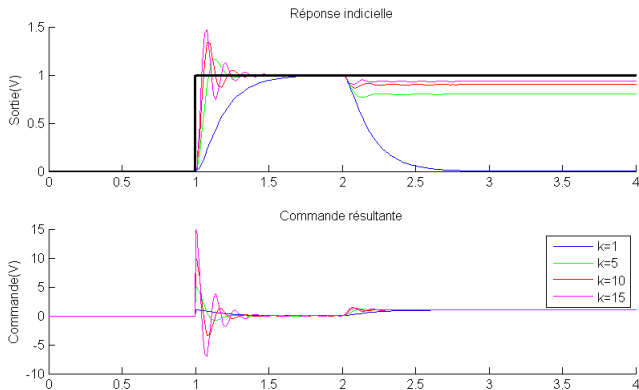
- Steady-State error  $\frac{\varepsilon}{Y_c}$  ?



# Preamble

## Introduction example 1

- Effect of  $k$  and a disturbance occurs at time  $t = 2$  seconds



## Question

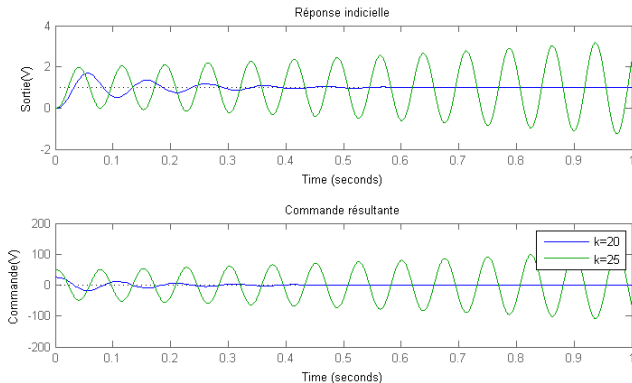
- Steady-State error  $\frac{\varepsilon}{D}$  ?



# Preamble

## Introduction example 1

- Effect of  $k$ , with  $k$  very large



## Question

- Unstability ?





### What to remember

- Effect of feedback and proportional action
- The steady-state error is small when  $k$  is huge, but the control action is important
- Danger : instability
- A simple proportional control  $k$  is (often) insufficient for the stability-precision trade-off

### Skills - supposed to be known

- Definition of a system in Matlab
- The final value theorem ! Beware of the area of convergence.
- Stability analysis (Routh criterion)



# Feedback configurations

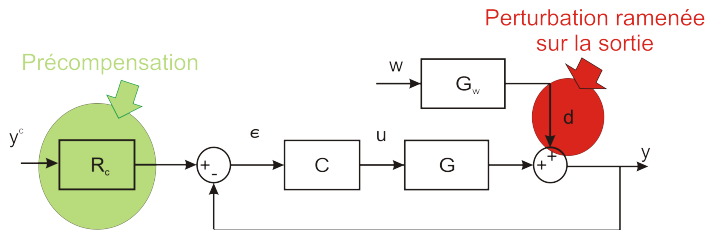
## Writing games and Schema

- In a general point of view, we can use 2 transfer functions in the control structure :

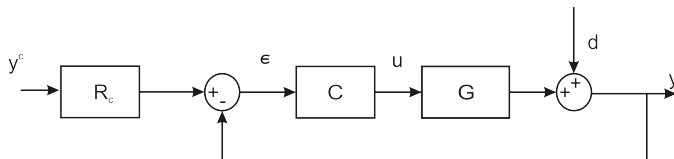
$$u = C_c(p)y^c - C(p)y$$

- Which can be rewritten into the RST structure :

$$u = \frac{T y^c - R y}{S}$$



**FIGURE – THE schéma**



- The transfer between the disturbance  $d$  and the error  $\varepsilon$  :

$$\varepsilon = \frac{-1}{1 + CG} d$$

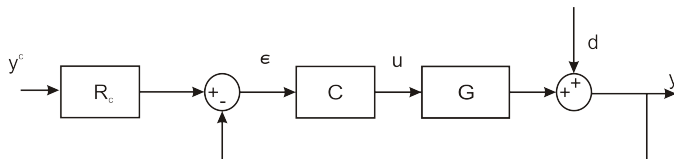
## Sensitivity : definition

$$S = \frac{1}{1 + CG}$$



# Sensitivities

## Model uncertainty sensitivity



- We can define the transfer between the reference  $y^c$  and the output  $y$  :

$$H = \frac{y}{y^c} = \frac{R_c C G}{1 + C G}$$

- What happens if a small variation  $\Delta G$  is applied on the model. What is  $\Delta H$ ?

## Sensitivity - property

$$\frac{\Delta H}{H} = S \frac{\Delta G}{G}$$

- We want to find  $C$  so that  $S$  is as small as possible !



# Sensitivities

## Complementary sensitivity

- Making  $S$  close to 0 is equivalent to making  $(1 - S)$  close to 1

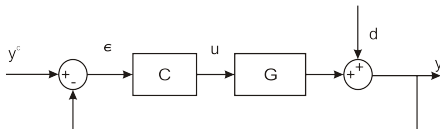
## Complementary sensitivity : definition

$$T = 1 - S = \frac{CG}{1 + CG}$$

- We have

$$H = \frac{R_c CG}{1 + CG} = TR_c$$

- The  $R_c$  pre-compensation is not necessarily required.
- We can therefore choose  $R_c = 1$  and we get the simplified schema :

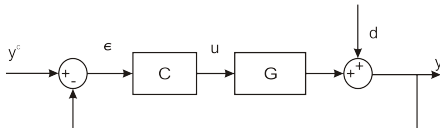




# Sensitivities

A major remark :  $R_C = 1$

- We often reason about



- We have  $\varepsilon = S(y^c - d)$
- Reference and disturbance play almost the same role on the deviation
- In the exercises : the main specifications are on  $\frac{y}{y^c}$  and therefore  $T$  but  $S$  is hidden ! In engineering problems, it is on  $S$  that the main specifications are imposed
- $R_C = 1$  is not absolute law ...

# Stability

## One definition among others

- Given the transfer  $G(p) = \frac{B(p)}{A(p)}$
- $A(p) = a_0 + a_1p + \dots + a_{n-1}p^{n-1} + p^n$



### Selected definition

A linear system is asymptotically stable if and only if its impulse response is absolutely integrable.

### Characterization

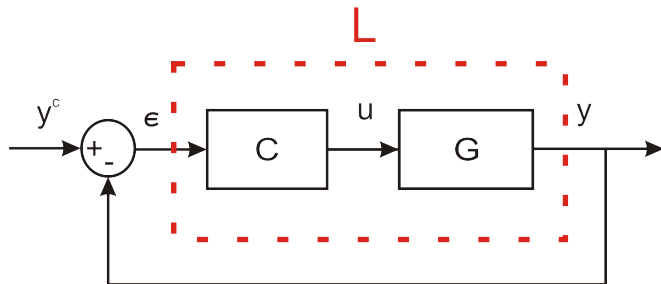
G is AS if and only if all its poles have a strictly negative real part.

- A tool already seen : the Routh criterion (see ST2 Modelisation)
- A useful tool... Matlab



# Stability

## Open Loop - Closed Loop Relationship



Relationship between  $L$  and  $\frac{1}{1+L}$







# Cauchy's theorem

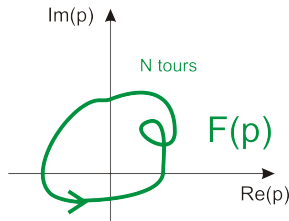
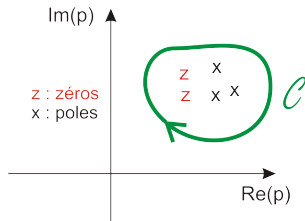
## Back to complex functions

- Let us consider  $F(p)$ , a meromorph complex function. Let us consider  $\mathcal{C}$  a closed contour.
- $Z$  : number of zeros of  $F$ ,  $P$  : number of poles of  $F$  inside the closed contour  $\mathcal{C}$

## Cauchy's theorem

- When  $p$  is moving on the contour  $\mathcal{C}$ ,  $F(p)$  describes a closed path
- $N$  : number of rotations of  $F(p)$  around 0, counted in the same direction of travel.

$$N = Z - P$$



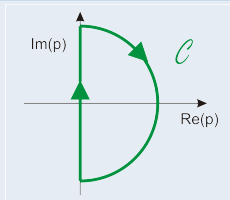


# Nyquist Criterion

## Cauchy's theorem application to stability analysis

- We look at the transfer  $\frac{1}{1+L}$
- Stability condition : no zeros with positive real part for  $(1 + L)$

### The contour of Bromwich



- The image of the Bromwich's contour by the  $1 + L(p)$  function must therefore do :  $-P$  turns around 0 !
- How to link this to  $L(p)$  and not  $1 + L(p)$  ?



# Nyquist Criterion

## Cauchy's theorem application to stability analysis : From $1 + L$ to $L$

- 1st observation :
  - The image of the Bromwich's contour by the  $1 + L(p)$  function must therefore do :  $-P$  turns around 0 !
  - Consequently, the image of the Bromwich's contour by the  $L(p)$  function must therefore do :  $-P$  turns around -1 ! (clockwise direction)

## Nyquist plot

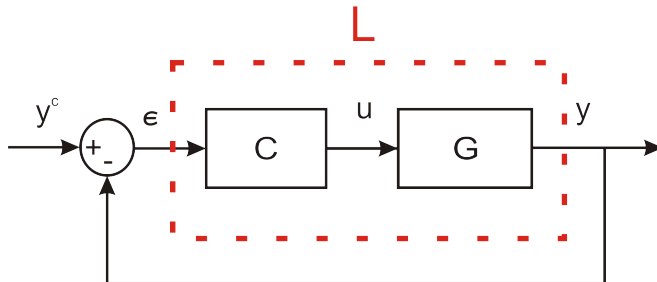
The Nyquist plot of  $L$  is the image of the Bromwich contour by the  $L(p)$  function.

- It is a closed curve.
- 2nd observation :
  - $1 + L$  and  $L$  have the same poles
  - $P$  is the number of poles of  $L$  with a positive real part



# Nyquist Criterion

The criterion, finally, we can get it !



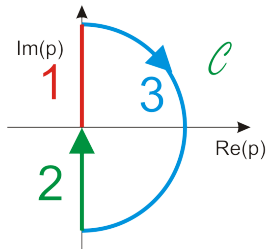
- Let us denote with  $P$  the number of poles of  $L(p)$  with a positive real part.

## Nyquist criterion

The transfer  $S = \frac{1}{1+L}$  is asymptotically stable if and only if the Nyquist plot of  $L$  encircles  $P$  times the point  $-1$  counter-clockwisely !

# Nyquist plot

## How to draw it ?



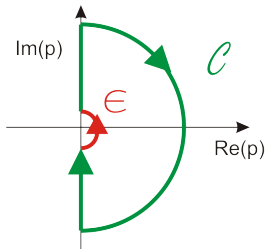
- Part 1 :  $p = jw$  : everything is provided by the Bode diagrams !
- Part 2 :  $p = -jw$  : it is the symmetrical of 1 with respect to the abscissa axis.
- Part 3 :  $p = Re^{i\Theta}$ , with  $R \rightarrow \infty$  : it's a point for any proper system. (this point is the origin if the system is strictly proper)



# Nyquist plot

## The little subtlety ...

- Cauchy's theorem : poles and zeros have to be strictly inside the contour.
- What if some poles/zeros are on the Bromwich contour ?



- We make indentations ... = we go around !
- The Nyquist plot is still a closed curve
- As a consequence : there are infinite phenomena happening on the Nyquist plot (half-turn, turn, 1 turn and a half, ...)



# My first Nyquist

To warm up

$$\frac{K}{1 + \tau p}$$



# My first Nyquist

To warm up, the opposite

$$\frac{-K}{1 + \tau p}$$







# My first Nyquist

To warm up, with a little help of Matlab

$$\frac{K}{1 + \tau p}, \quad \frac{-K}{1 + \tau p}$$

## The code

### Model :

- `k=5 ; tau=1 ;`
- `p=tf('p');`
- `sys=k/(1+tau*p); sys2=-k/(1+tau*p);`

### Plots :

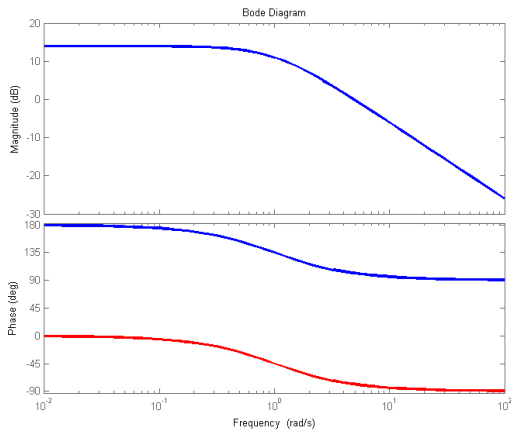
- `figure`
- `bode(sys,'r',sys2,'b')`
- `nyquist(sys,'r',sys2,'b')`



# My first Nyquist

To warm up, with a little help of Matlab

$$\frac{K}{1 + \tau p}, \quad \frac{-K}{1 + \tau p}$$

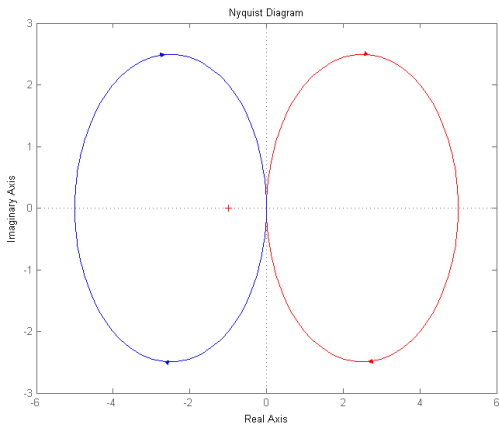




# My first Nyquist

To warm up, with a little help of Matlab

$$\frac{K}{1 + \tau p}, \quad \frac{-K}{1 + \tau p}$$





# Nyquist for system with integral action

Indentation ... Bypass ! Bypass ! Half turn

$$\frac{K}{p(1 + \tau p)}$$



# Nyquist for system with integral action

Indentation ... Bypass ! Bypass ! Half turn, the opposite

$$\frac{-K}{p(1 + \tau p)}$$



# Nyquist for system with double integral action

Indentation ... Bypass ! Bypass ! Full turn

$$\frac{K}{p^2(1 + \tau p)}$$

# Nyquist - the trap

A matter of good direction

$$\frac{K}{(1-p)(2+p)}$$

$$\frac{-K}{(1-p)(2+p)}$$



# Nyquist - the trap

A matter of good direction

$$\frac{K}{(1+p)(2-p)}$$

$$\frac{-K}{(1+p)(2-p)}$$





# Nyquist - A last one for the road

## The case of the oscillator

$$\frac{K}{(p+2)(p^2+4)}$$





## Expected skills

- Impact of the closed loop on sensitivity
- Drawing the Nyquist plot for a given transfer
- Determine its stability in CL using the Nyquist criterion