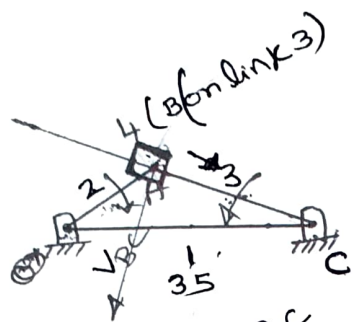


Numbered on Coriolis Component of Acceleration.



Link 2 rotates at $20 \text{ rad/sec} = \omega_2$

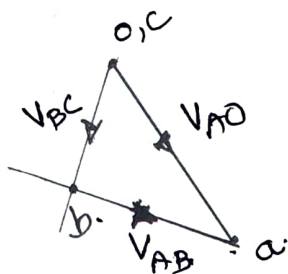
$$OC = 35 \text{ cm.}$$

$$OA = 15 \text{ cm.}$$

$$BC = 25 \text{ cm.}$$

Draw velocity and acceleration diagram

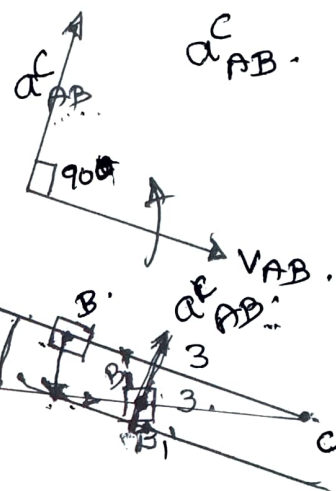
91



velocity diagram.

By measurement.

$$\left. \begin{aligned} V_{AB} &= 2.6 \text{ m/s.} \\ V_{BC} &= 1.8 \text{ m/s.} \end{aligned} \right\} \text{ measurement}$$



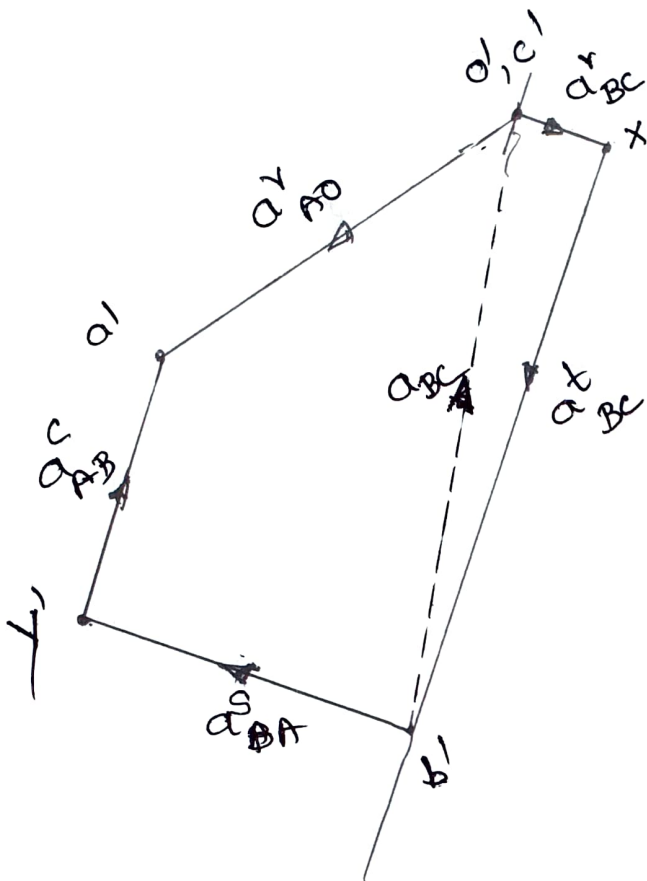
(OA) Link 2 rotates clockwise $V_{AO} \perp OA$

$$V_{AO} = \omega_2 \times OA = 20 \text{ rad/sec} \times 0.15 \text{ m.}$$

$$= 3 \text{ m/s.}$$

Acceleration Diagram

$$a_{AO} = \frac{V_{AO}^2}{AO} = \frac{3^2}{0.15} = 60 \text{ m/s}^2$$



$$a_{AB}^c = 2 V_{AB} \times \omega_3$$

$$= 2 \times 2.6 \times \omega_3$$

$$\omega_3 = \frac{V_{BC}}{BC} = \frac{1.8}{.25}$$

$$= 7.2 \text{ rad/sec}$$

$$a_{AB}^c = 2 \times 2.6 \times 7.2$$

$$= \underline{37.44 \text{ m/sec}^2}$$

$$a_{BC}^r = \frac{V_{BC}^2}{BC} = \frac{1.8^2}{.25}$$

$$= 12.96 \text{ m/s}^2$$

Acceleration diagram $a_{AB}^s = \text{sliding}$.

$$a_{BA}^s = \rightarrow$$

with the slider. Link 1 is pivoted at O.

Example 2.23 A quick return mechanism is shown in Fig.2.41(a). Link 2 rotates at 20 rad/sec. Draw the velocity and acceleration diagram. } BC225

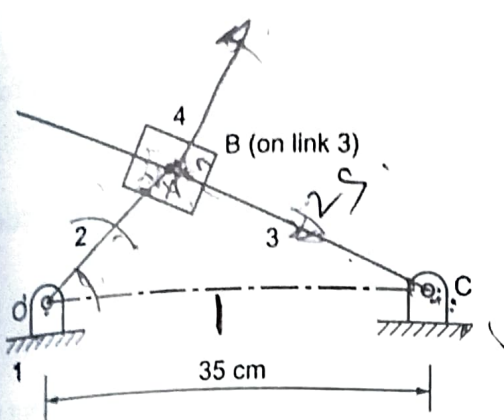
☺ **Solution : Given Data :** The angular velocity of link 2

$$\omega_2 = 20 \text{ rad/sec ;}$$

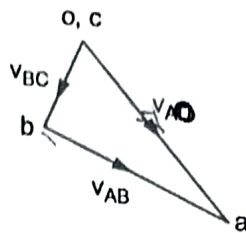
$$OA = 15 \text{ cm} = 0.15 \text{ m ;}$$

$$v_{AO} = \omega_2 \cdot OA$$

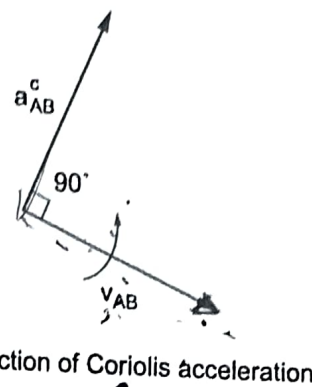
$$= 20 \times 0.15 = 3 \text{ m/sec}$$



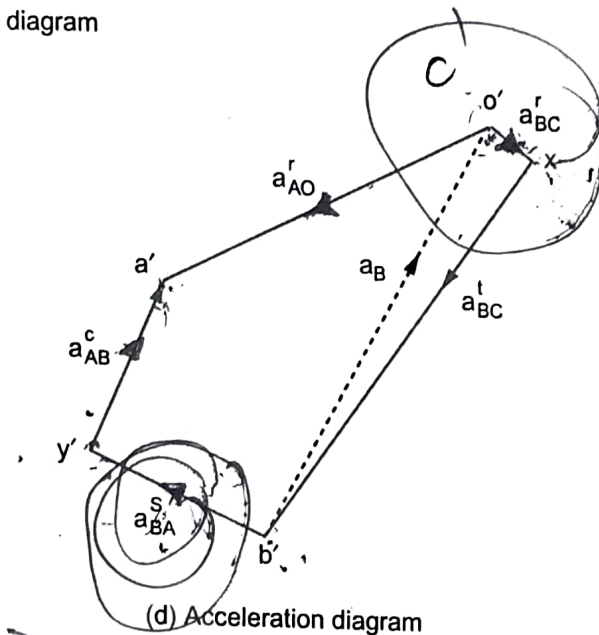
(a) Configuration diagram



(b) Velocity diagram



(c) Direction of Coriolis acceleration



(d) Acceleration diagram

Fig. 2.41.

Procedure :

(a) **Configuration Diagram** : Refer Fig.2.41(a).

(b) **Velocity Diagram** : Refer Fig.2.41(b).

1. The velocity of point A with respect to O, v_{AO} is perpendicular to OA. Since O and C are fixed points on the configuration diagram, they may be taken as one point o or c on the velocity diagram. From o draw vector oa representing v_{AO} .
2. The velocity of point B on link 3 (CB with respect to C, v_{BC} is perpendicular to CB with unknown magnitude. From c draw vector cb representing v_{BC} . It contains point b.
3. The velocity of point B with respect to A, v_{BA} is along the path of the slider. From 'a' draw a line ab parallel to the path of slider. It intersects ob at o. By measurement $ob = v_{BO} = v_B = 1.5$ m/sec.

(c) **Acceleration Diagram** : Refer Fig.2.41(d).

1. Draw $o'a' = a_{AO}^r = \frac{v_{AO}^2}{OA} = \frac{(3)^2}{0.15} = 60$ m/sec² parallel to AO with some scale.

B-coincident pt on
Link 3.

B & C T, R, C
with reference to
A only.

2. From o' draw vector $o'x' = a_{BC}^r = \frac{v_{BC}^2}{BC} = \frac{(1.5)^2}{0.25} = 9 \text{ m/sec}^2$ parallel to CB. From x draw $xb' = a_{BC}^t$, not known in magnitude and perpendicular to $o'x$.
3. Coriolis acceleration a_{AB}^c is introduced in the problem. Its magnitude and direction both are to be determined.

$$\text{Magnitude } a_{AB}^c = 2 \cdot \omega_3 \cdot v_{AB}$$

$$\omega_3 = \frac{v_{BC}}{BC} = \frac{1.5}{0.25} = 6 \text{ rad/sec in anticlockwise direction about C.}$$

$$v_{AB} = 2.64 \text{ m/sec}$$

$$a_{AB}^c = 2 \times 6 \times 2.64 = 31.68 \text{ m/sec}^2$$

where ω_3 is the angular velocity of link 3.

Direction : a_{AB}^c is perpendicular to v_{AB} . It is shown in Fig.2.41(c). Rotation of link 3 is anticlockwise. Thus vector v_{AB} is rotated by 90° in anticlockwise direction to find the direction of a_{AB}^r .

[\because Link 3 rotates in anticlockwise sense about C].

4. The acceleration of sliding of B relative to A, a_{BA}^s is parallel to link 3. Its magnitude is unknown.
5. From a' draw vector $a'y' = 31.68 \text{ m/sec}^2 = a_{AB}^{cr}$.
6. From y' draw vector $y'b'$ representing a_{BA}^s parallel to link 3. It intersects xb' at point b' .
7. Join o' to b' . Vector $o'b'$ represents the acceleration of B with respect to O. Thus $a_B = o'b'$.