## Problem 4

A schematic diagram of a rotor mounted in bearings is shown in Fig. 4. The moment of inertia of the rotor about the axis of rotation is  $J=0.31~kgm^2$ . Let us assume that at t=0 the rotor is rotating at the angular velocity  $\omega(0)=\omega_0=121~\frac{rad}{sec}$ . We also assume that the friction in bearings is viscous friction, where the coefficient of friction is  $b=0.02~\frac{Nm}{rad}$ .

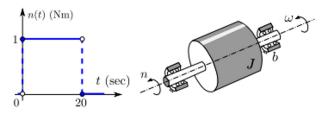


Figure 4: Input signal nad rotor mounted in bearings

Assume that the input to the system is the external applied torque n(t) (Nm) in the form of a pulse signal shown in Fig. 4, while the output is the angular velocity  $\omega(t)$   $(\frac{rad}{s})$ . Find and plot free  $y_{free}$ , forced  $y_{forced}$ , and total response  $y_{total}$  of the system. Calculate the total response  $y_{total}$  for time t=20 s.

Pforced: 
$$J(s) = +6Y(s) - U_0 = \frac{1 - e^{-20s}}{s}$$

$$Y(s) (Js+6) = \frac{1 - e^{-20s}}{s}$$

$$Y(s) = \frac{1 - e^{-20s}}{s(Js+6)} = \frac{1}{J} (\frac{1 - e^{-20s}}{s+\frac{1}{J}})$$

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syms t;
        n(t) = heaviside(t)-heaviside(t-20);
        n(s) = laplace(n(t),t,s);
        Yfree(s)=121*0.31/(0.31*s+0.02);
        Yforced(s) = n(s)/(0.31*s+0.02);
11 -
12 -
        yfree(t) = ilaplace(Yfree(s),s,t);
       yforced(t) = ilaplace(Yforced(s),s,t);
13
        disp(vfree(t));
16
        ytotal(t) = yfree(t)+yforced(t);
18
19 -
       disp(ytotal(t));
20
       disp(eval(ytotal(20)));
22
23 -
        fplot(yfree(t), [0,100]);
        title('Free response');
xlabel('Time[s]');
        ylabel('Ang. Velocity [rad/s]');
        fplot(yforced(t), [0,100]);
```

