



Xi'an Jiaotong-Liverpool University  
西交利物浦大學

# EEE220 Instrumentation and Control System

*2018-19 Semester 2*

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# Lecture 11

# Outline

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## State Variable Models: part 1/2

- ☐ Introduction
- ☐ State Variables
- ☐ State-space Modeling
- ☐ State Space Representation in Matrix Form
- ☐ Time-domain response (Solution of State-space Models)
- ☐ Conversion between State-space Model and Transfer Function
- ☐ Analysis of the State-space Models using Matlab

# Introduction

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In the preceding classes, we studied several useful approaches to the analysis of design of control system including (suppose the system order is  $n$ ):

- **Ordinary differential equations (ODEs)**:  $n$ th-order differential equations; time-domain description of system;
- **Transfer function**: Laplace transform of ODEs; frequency-domain description;
- **Block diagram & signal-flow graph**: graphic ways derived from transfer functions.

In this chapter, we'll learn

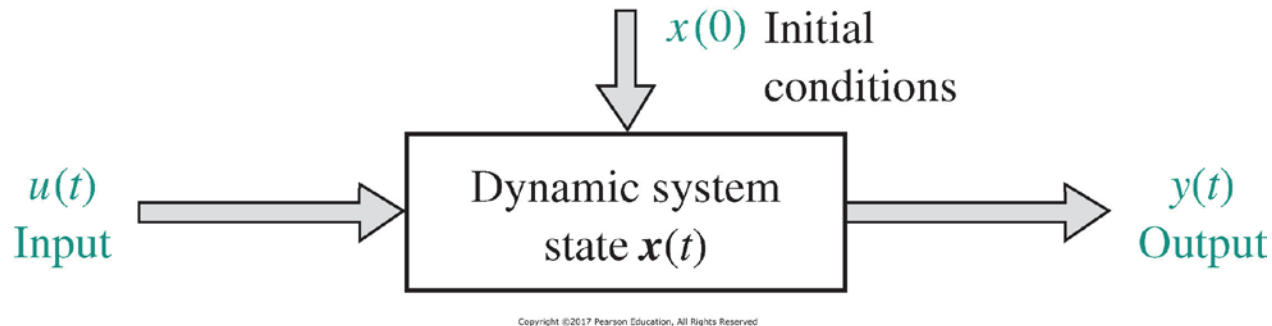
- **State-space model** (or **state variable model**) which
  - is a set of **1<sup>st</sup> order** differential equations therefore a time-domain description of a system and can be further represented in a convenient **matrix-vector** form;
  - powerful mathematical & computational tools available for analyzing and designing a system using state-space model;
  - can be extended to **nonlinear, time-varying & multiple input-output** system.

**State-space model is an essential basis for modern control theory.**

# State of A System

The **State** of a system is a set of variables whose values, together with the input signals and the equations describing the dynamics, will provide the future state and output of the system;

It is the minimum information needed about the system in order to determine its future behavior.

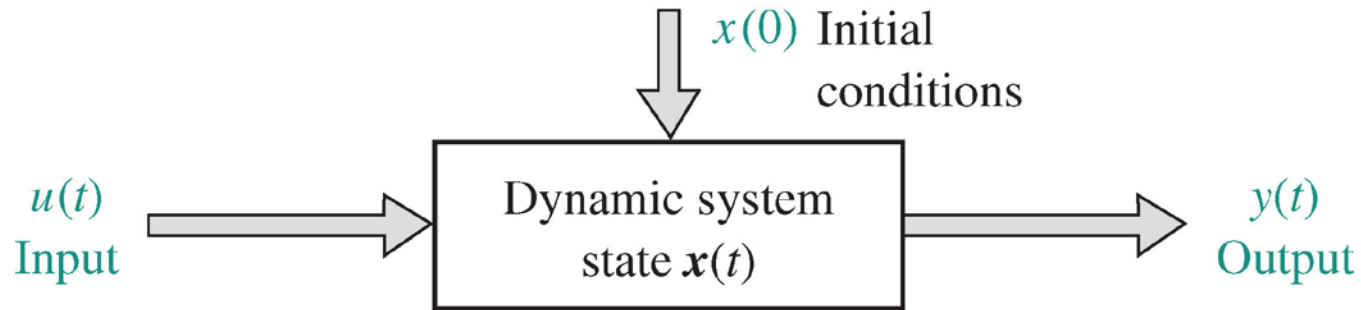


State variables:  $[x_1(t), x_2(t), x_3(t), \dots x_n(t)]$

Input signals:  $[u_1(t), u_2(t), \dots u_m(t)]$

Output signals:  $[y_1(t), y_2(t), \dots y_p(t)]$

# State Variables



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- The **state variables** of a system are defined as a minimal set of variables, which are set of variables of smallest possible size that together with any input to the system is sufficient to determine the future behavior (i.e., output) of the system.
- At any initial time  $t = t_0$ , the state variables  $x_1(t_0), x_2(t_0), x_3(t_0), \dots, x_n(t_0)$  defines the initial states of the system;
- Once the inputs of the system for  $t \geq t_0$  and the initial states just defined are specified, the state variables should completely define the future behavior of the system.

# State-Space Modeling

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- **State differential equations:** first-order differential equations written in terms of the state variables ( $x_1(t), x_2(t), \dots, x_n(t)$ ) and the input ( $u_1(t), u_2(t), \dots, u_m(t)$ );
- A **state-space model** represents a system by a series of first-order differential state equations and algebraic output equations; It simplifies analysis of complex systems with multiple inputs and outputs;
- State-space models are numerically efficient to solve, can handle complex systems, allow for a more geometric understanding of systems, and form the basis of modern control theory.

# Example 11.1

Consider the following system where  $u(t)$  is the input and  $\dot{x}(t)$  is the output:

$$\ddot{x} + 5\dot{x} + 3x + 2x = u$$

## How to generate state-space model?

- Changing variables, let  $x_1 = x$ ,  $x_2 = \dot{x}$ ,  $x_3 = \ddot{x}$ ,

$$\ddot{x} = -5\dot{x} - 3x + u \quad \longrightarrow \quad \left\{ \begin{array}{l} \dot{x}_3 = -5x_3 - 3x_2 - 2x_1 + u \\ \dot{x}_2 = x_3 \\ \dot{x}_1 = x_2 \end{array} \right.$$

**State equations**

- Obtaining output:

$$y = \dot{x} = x_2 \quad \left\} \quad \textbf{Output equation}$$

**The system has 1 input ( $u(t)$ ), 1 output ( $y(t)$ ) and 3 state variables ( $x_1(t)$ ,  $x_2(t)$ ,  $x_3(t)$ ).**



# State-Space Modeling: General Form

General state-space models have the following form:

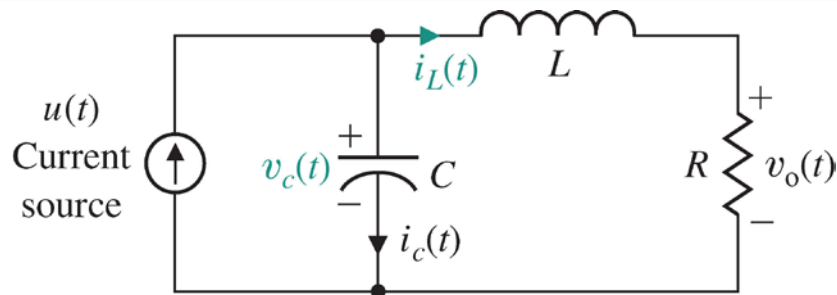
**State equations**

$$\left\{ \begin{array}{l} \dot{x}_1 = f_1(x_1, x_2, \dots, x_n, u_1, u_2, \dots, u_m, ) \\ \dot{x}_2 = f_2(x_1, x_2, \dots, x_n, u_1, u_2, \dots, u_m, ) \\ \vdots \\ \dot{x}_n = f_n(x_1, x_2, \dots, x_n, u_1, u_2, \dots, u_m, ) \end{array} \right.$$

**Output equations**

$$\left\{ \begin{array}{l} y_1 = h_1(x_1, x_2, \dots, x_n, u_1, u_2, \dots, u_m, ) \\ y_2 = h_2(x_1, x_2, \dots, x_n, u_1, u_2, \dots, u_m, ) \\ \vdots \\ y_p = h_p(x_1, x_2, \dots, x_n, u_1, u_2, \dots, u_m, ) \end{array} \right.$$

# Example 11.2: RLC Circuit



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Ordinary Differential Equations:  $i_c(t) = C \frac{dv_c}{dt} = u(t) - i_L(t)$

$$L \frac{di_L}{dt} = -Ri_L(t) + v_c(t)$$

Output:  $v_o(t) = Ri_L(t)$

- Choose variables, let  $x_1(t) = v_c(t)$ ,  $x_2(t) = i_L(t)$

$$\frac{dx_1}{dt} = -\frac{1}{C}x_2(t) + \frac{1}{C}u(t)$$

$$\frac{dx_2}{dt} = \frac{1}{L}x_1(t) - \frac{R}{L}x_2(t)$$

$$y(t) = v_o(t) = Rx_2(t)$$

$$\dot{x}_1 = -\frac{1}{C}x_2 + \frac{1}{C}u$$

$$\dot{x}_2 = \frac{1}{L}x_1 - \frac{R}{L}x_2$$

$$y = Rx_2$$

The state variables describing a system are **not a unique set**, several alternative sets of state variables can be chosen.

# State Variables Selection

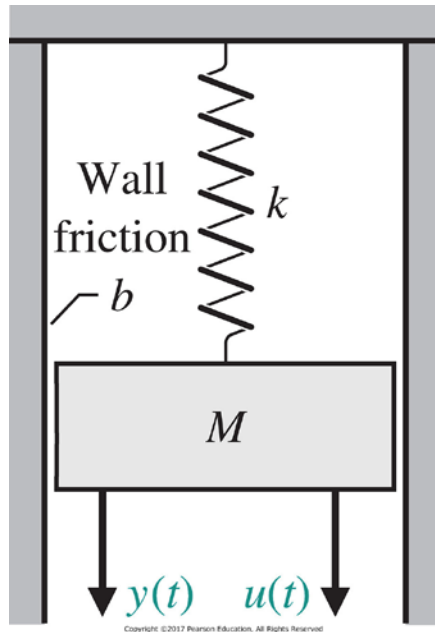
**State variables are typically associated with energy storage:**

- Each state Variable has memory;
- Each state variable has an “initial condition”;

**For a system, the number of state variables required is equal to the number of **independent energy-storage** elements.**

- In electric circuits, the energy storage devices are the **capacitors** and **inductors**. They contain all of the state information or “memory” in the system. State variables:
  - Voltage across capacitors
  - Current through inductors
- In mechanical systems, energy is stored in **springs** and **masses**. State variables:
  - Spring displacement
  - Mass position and velocity
- **Resistors** (in electric circuits) and **dampers** (mechanical systems) are energy dissipaters, **they don't store energy**.

# Example 11.3: Spring-Mass-Damper System



Independent energy-storage elements:

- 1 spring;
- 1 mass.

Number of state variables: 2

- let  $x_1 = y(t)$ ,  $x_2 = \dot{x}_1$ , therefore,  $x_1$  and  $x_2$  will represent displacement and velocity respectively.

$$M \frac{d^2 y(t)}{dt^2} + b \frac{dy(t)}{dt} + ky(t) = u(t)$$



$$\dot{x}_2 = -\frac{b}{M} x_2 - \frac{k}{M} x_1 + \frac{1}{M} u$$

$$\dot{x}_1 = x_2$$

Output:

$$y = x_1$$

# State-space Representation in Matrix Form

## State Equations:

$$\dot{x}_1(t) = a_{11}x_1(t) + a_{12}x_2(t) + \cdots + a_{1n}x_n(t) + b_{11}u_1(t) + \cdots + b_{1m}u_m(t)$$

$$\dot{x}_2(t) = a_{21}x_1(t) + a_{22}x_2(t) + \cdots + a_{2n}x_n(t) + b_{21}u_1(t) + \cdots + b_{2m}u_m(t)$$


$\vdots$

$$\dot{x}_n(t) = a_{n1}x_1(t) + a_{n2}x_2(t) + \cdots + a_{nn}x_n(t) + b_{n1}u_1(t) + \cdots + b_{nm}u_m(t)$$

$$\mathbf{x}(t) = \begin{pmatrix} x_1(t) \\ x_2(t) \\ \vdots \\ x_n(t) \end{pmatrix} \quad \mathbf{u}(t) = \begin{pmatrix} u_1(t) \\ u_2(t) \\ \vdots \\ u_m(t) \end{pmatrix}$$

$$\mathbf{y}(t) = \begin{pmatrix} y_1(t) \\ y_2(t) \\ \vdots \\ y_p(t) \end{pmatrix}$$

## Output Equations:


$$\dot{\mathbf{x}}(t) = \mathbf{A}\mathbf{x}(t) + \mathbf{B}\mathbf{u}(t)$$

$$\mathbf{y}(t) = \mathbf{C}\mathbf{x}(t) + \mathbf{D}\mathbf{u}(t)$$

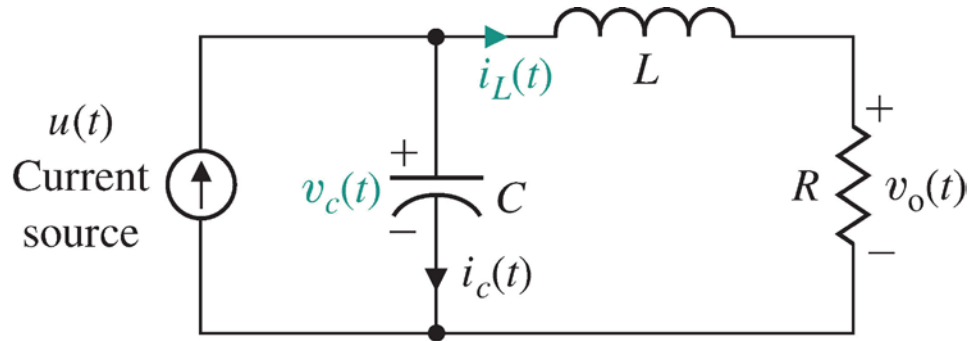
*Dimensions:*  $\mathbf{x}(n \times 1), \mathbf{u}(m \times 1), \mathbf{y}(p \times 1);$   
 $\mathbf{A}(n \times n), \mathbf{B}(n \times m), \mathbf{C}(p \times n), \mathbf{D}(p \times m).$

*\*note: boldface small letter: vector; boldface capital letter: matrix.*

# Examples

$$\dot{\mathbf{x}}(t) = \mathbf{A}\mathbf{x}(t) + \mathbf{B}u(t)$$

$$\mathbf{y}(t) = \mathbf{C}\mathbf{x}(t) + \mathbf{D}u(t)$$



$$\begin{aligned}\dot{x}_1 &= -\frac{1}{C}x_2 + \frac{1}{C}u \\ \dot{x}_2 &= \frac{1}{L}x_1 - \frac{R}{L}x_2 \\ y &= Rx_2\end{aligned}$$

$$A = \begin{bmatrix} 0 & -\frac{1}{C} \\ \frac{1}{L} & -\frac{R}{L} \end{bmatrix},$$

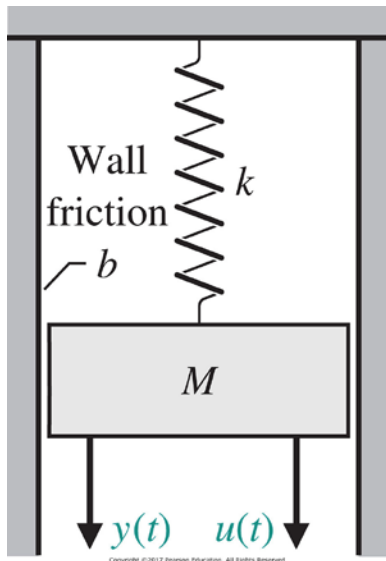
$$B = \begin{bmatrix} \frac{1}{C} \\ 0 \end{bmatrix}$$

$$C = [0 \quad R],$$

$$D = [0]$$

$$\dot{\mathbf{x}}(t) = \mathbf{A}\mathbf{x}(t) + \mathbf{B}u(t)$$

$$\mathbf{y}(t) = \mathbf{C}\mathbf{x}(t) + \mathbf{D}u(t)$$



$$\dot{x}_1 = x_2$$

$$\dot{x}_2 = \frac{-b}{M}x_2 - \frac{k}{M}x_1 + \frac{1}{M}u$$

$$y = x_1$$

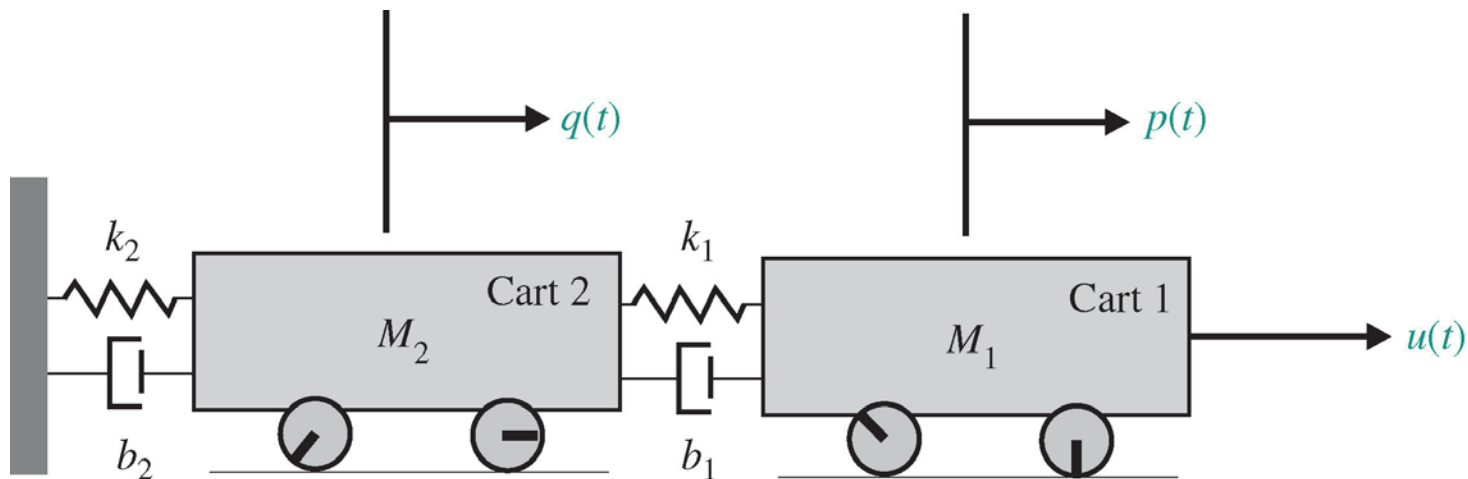
$$A = \begin{bmatrix} 0 & 1 \\ -\frac{k}{M} & -\frac{b}{M} \end{bmatrix},$$

$$B = \begin{bmatrix} 0 \\ \frac{1}{M} \end{bmatrix}$$

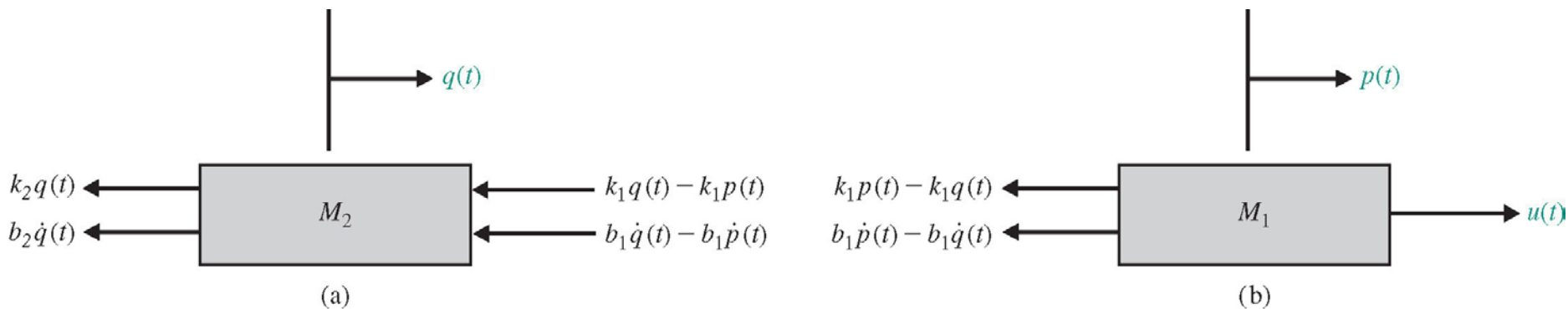
$$C = [1 \quad 0],$$

$$D = [0]$$

# Example 11.4: Two Rolling Carts

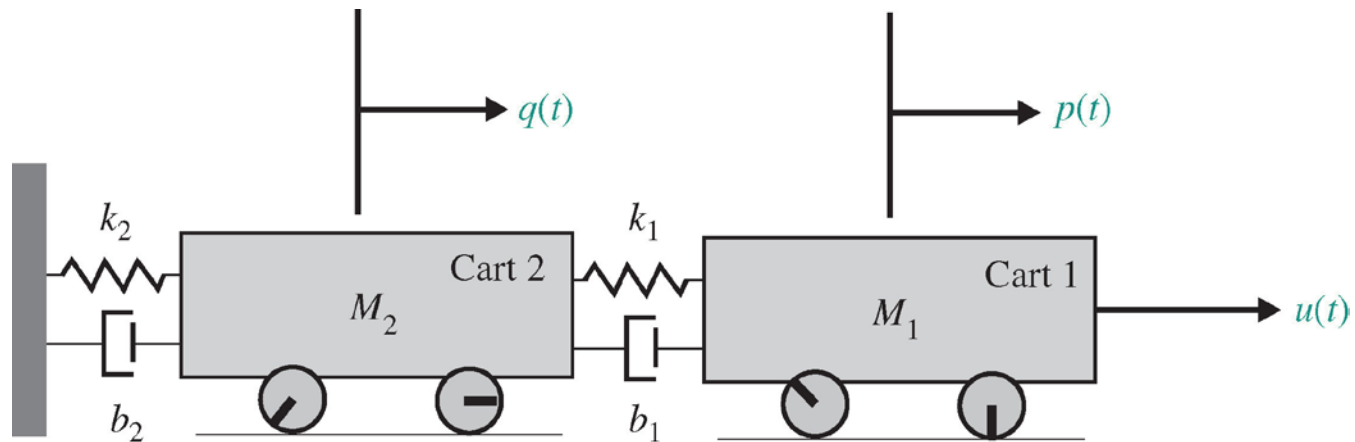


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where  $p$ ,  $q$  are position of carts;  $M_1$  and  $M_2$  are mass of carts;  
 $k_1$ ,  $k_2$ : spring coefficients;  $b_1$ ,  $b_2$ : damper coefficients.  $u$ : external force.

For Mass 1:

$$M_1 \ddot{p} + b_1 \dot{p} + k_1 p = u + k_1 q + b_1 \dot{q},$$

For Mass 2:

$$M_2 \ddot{q} = k_1(p - q) + b_1(\dot{p} - \dot{q}) - k_2 q - b_2 \dot{q},$$

**How many state variables required?**

$$\dot{\mathbf{x}} = \mathbf{Ax} + \mathbf{Bu}$$

$$\begin{aligned} x_1 &= p, & x_3 &= \dot{x}_1 = \dot{p}, \\ x_2 &= q, & x_4 &= \dot{x}_2 = \dot{q}. \end{aligned} \quad \mathbf{x} = \begin{pmatrix} x_1 \\ x_2 \\ x_3 \\ x_4 \end{pmatrix} = \begin{pmatrix} p \\ q \\ \dot{p} \\ \dot{q} \end{pmatrix}$$

$$\dot{x}_3 = \ddot{p} = -\frac{b_1}{M_1}\dot{p} - \frac{k_1}{M_1}p + \frac{1}{M_1}u + \frac{k_1}{M_1}q + \frac{b_1}{M_1}\dot{q},$$

$$\dot{x}_4 = \ddot{q} = -\frac{k_1 + k_2}{M_2}q - \frac{b_1 + b_2}{M_2}\dot{q} + \frac{k_1}{M_2}p + \frac{b_1}{M_2}\dot{p},$$

$$\mathbf{A} = \begin{bmatrix} 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 1 \\ -\frac{k_1}{M_1} & \frac{k_1}{M_1} & -\frac{b_1}{M_1} & \frac{b_1}{M_1} \\ \frac{k_1}{M_2} & -\frac{k_1 + k_2}{M_2} & \frac{b_1}{M_2} & -\frac{b_1 + b_2}{M_2} \end{bmatrix}, \quad \text{and} \quad \mathbf{B} = \begin{bmatrix} 0 \\ 0 \\ \frac{1}{M_1} \\ 0 \end{bmatrix},$$

$$y = [1 \quad 0 \quad 0 \quad 0]\mathbf{x} = \mathbf{Cx}.$$

$$\mathbf{D} = [0]$$

# Solution of State Equations

Consider the first-order differential equation (**one-dimension**):

$$\dot{x} = ax + bu$$

With Laplace transform,

$$sX(s) - x(0) = aX(s) + bU(s)$$

$$X(s) = \frac{x(0)}{s - a} + \frac{b}{s - a} U(s)$$

Inverse Laplace transform satisfies, note :  $\mathcal{L}^{-1}(f \cdot g) = \mathcal{L}^{-1}(f) * \mathcal{L}^{-1}(g)$  :

$$x(t) = e^{at}x(0) + be^{at} * u(t)$$

Therefore:

$$x(t) = e^{at}x(0) + \int_0^t e^{a(t-\tau)} bu(\tau) d\tau$$

# Solutions for Matrix Form

$$\dot{\mathbf{x}}(t) = \mathbf{A}\mathbf{x}(t) + \mathbf{B}\mathbf{u}(t)$$

$$s\mathbf{X}(s) - \mathbf{x}(0) = \mathbf{A}\mathbf{X}(s) + \mathbf{B}\mathbf{U}(s),$$

$$\mathbf{X}(s) = [s\mathbf{I} - \mathbf{A}]^{-1}\mathbf{x}(0) + [s\mathbf{I} - \mathbf{A}]^{-1}\mathbf{B}\mathbf{U}(s)$$

And we let  $[s\mathbf{I} - \mathbf{A}]^{-1} = \Phi(s)$ , it is the Laplace form of

$\Phi(t) = \exp(\mathbf{A}t)$  ---- *fundamental/state transition matrix,  
Describes the unforced response of the system (when  $u(t)=0$ ).*

The matrix exponential function is defined as:

$$e^{\mathbf{A}t} = \exp(\mathbf{A}t) = \mathbf{I} + \mathbf{A}t + \frac{\mathbf{A}^2 t^2}{2!} + \cdots + \frac{\mathbf{A}^k t^k}{k!} + \cdots$$

Then:

$$\mathbf{x}(t) = \Phi(t)\mathbf{x}(0) + \int_0^t \Phi(t - \tau)\mathbf{B}\mathbf{u}(\tau) d\tau$$

Solution to the matrix:

$$\mathbf{x}(t) = \exp(\mathbf{A}t) \mathbf{x}(0) + \int_0^t \exp[\mathbf{A}(t - \tau)] \mathbf{B}\mathbf{u}(\tau) d\tau$$

# Fundamental/State Transition Matrix

$$\mathbf{x}(t) = \mathbf{\Phi}(t)\mathbf{x}(0) + \int_0^t \mathbf{\Phi}(t - \tau)\mathbf{B}\mathbf{u}(\tau) d\tau$$

$$\mathbf{\Phi}(t) = \exp(\mathbf{A}t)$$

The solution to the unforced system (when  $\mathbf{u}(t)=0$ ):

$$\begin{pmatrix} x_1(t) \\ x_2(t) \\ \vdots \\ x_n(t) \end{pmatrix} = \begin{bmatrix} \phi_{11} & \dots & \phi_{1n} \\ \phi_{12} & \dots & \phi_{2n} \\ \vdots & & \vdots \\ \phi_{n1} & \dots & \phi_{nn} \end{bmatrix} \begin{pmatrix} x_1(0) \\ x_2(0) \\ \vdots \\ x_n(0) \end{pmatrix}$$

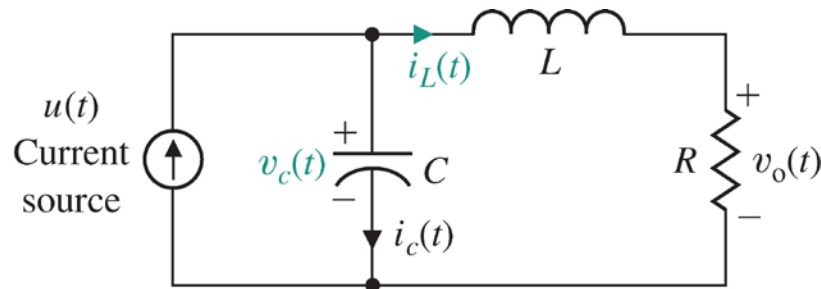
–  $\phi_{ij}$  is the response of the  $i$ th state variable due to an initial condition on the  $j$ th state variable when there are zero initial conditions on all the other variables.

# Example 11.5: Time Response of Unforced System

$$\underline{\mathbf{x}}(t) = \underline{\Phi}(t)\underline{\mathbf{x}}(0)$$

$$[s\mathbf{I} - \mathbf{A}]^{-1} = \underline{\Phi}(s)$$

$$\underline{\Phi}(t) = \exp(\mathbf{A}t) = \mathcal{L}^{-1}(\underline{\Phi}(s)) \quad \text{--- fundamental/state transition matrix}$$



$$\mathbf{A} = \begin{bmatrix} 0 & -\frac{1}{C} \\ \frac{1}{L} & -\frac{R}{L} \end{bmatrix},$$

$$\mathbf{C} = [0 \quad R],$$

$$\mathbf{B} = \begin{bmatrix} \frac{1}{C} \\ 0 \end{bmatrix}$$

$$\mathbf{D} = [0]$$

Assume  $C = \frac{1}{2}$ ,  $L = 1$ ,  $R = 3$ , then:

$$\mathbf{A} = \begin{bmatrix} 0 & -2 \\ 1 & -3 \end{bmatrix} \longrightarrow s\mathbf{I} - \mathbf{A} = \begin{bmatrix} s & 2 \\ -1 & s+3 \end{bmatrix}$$

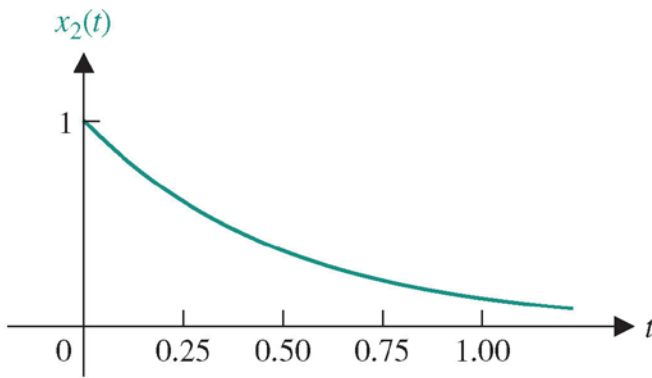
$$\underline{\Phi}(s) = [s\mathbf{I} - \mathbf{A}]^{-1} = \frac{\text{adj}\left(\begin{bmatrix} s & 2 \\ -1 & s+3 \end{bmatrix}\right)}{\det\left(\begin{bmatrix} s & 2 \\ -1 & s+3 \end{bmatrix}\right)} = \begin{bmatrix} \frac{s+3}{s^2+3s+2} & \frac{-2}{s^2+3s+2} \\ \frac{1}{s^2+3s+2} & \frac{s}{s^2+3s+2} \end{bmatrix}$$

$$\begin{aligned}
 \Phi(t) = \mathcal{L}^{-1}(\Phi(s)) &= \mathcal{L}^{-1}\left(\begin{bmatrix} \frac{s+3}{s^2+3s+2} & \frac{-2}{s^2+3s+2} \\ \frac{1}{s^2+3s+2} & \frac{s}{s^2+3s+2} \end{bmatrix}\right) \\
 &= \begin{bmatrix} \left(\frac{2}{s+1} + \frac{-1}{s+2}\right) & \left(\frac{-2}{s+1} + \frac{2}{s+2}\right) \\ \left(\frac{1}{s+1} + \frac{-1}{s+2}\right) & \left(\frac{-1}{s+1} + \frac{2}{s+2}\right) \end{bmatrix} \\
 &= \begin{bmatrix} (2e^{-t} - e^{-2t}) & (-2e^{-t} + 2e^{-2t}) \\ (e^{-t} - e^{-2t}) & (-e^{-t} + 2e^{-2t}) \end{bmatrix}
 \end{aligned}$$

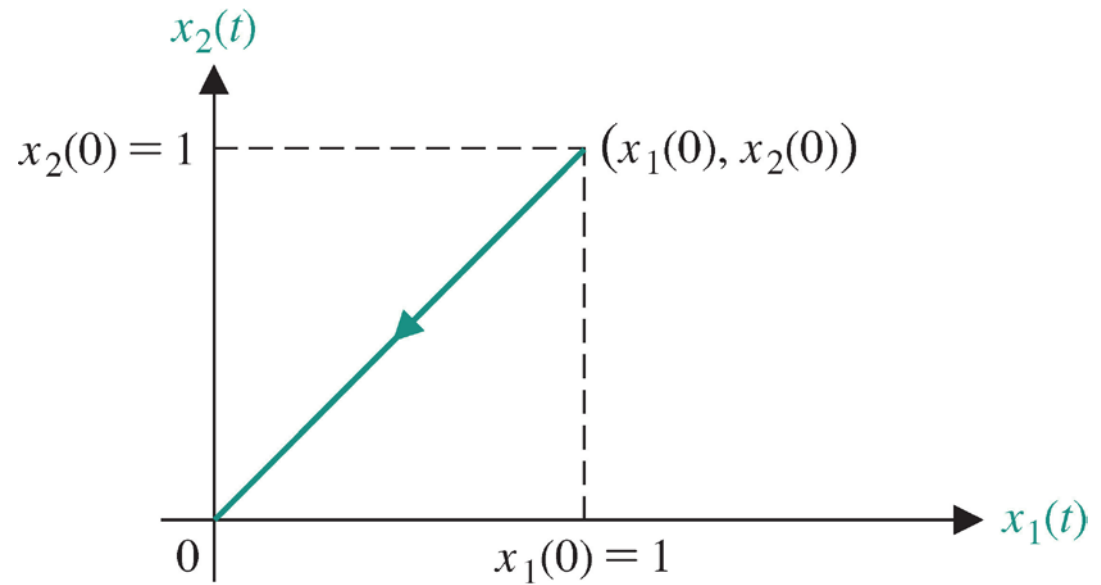
Assume  $x_1(0) = x_2(0) = 1$ :

$$\begin{pmatrix} x_1(t) \\ x_2(t) \end{pmatrix} = \begin{bmatrix} (2e^{-t} - e^{-2t}) & (-2e^{-t} + 2e^{-2t}) \\ (e^{-t} - e^{-2t}) & (-e^{-t} + 2e^{-2t}) \end{bmatrix} \begin{pmatrix} 1 \\ 1 \end{pmatrix} = \begin{bmatrix} e^{-2t} \\ e^{-2t} \end{bmatrix}$$

$$\begin{pmatrix} x_1(t) \\ x_2(t) \end{pmatrix} = \begin{bmatrix} e^{-2t} \\ e^{-2t} \end{bmatrix}$$



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How about  $y(t)$ ?



# Quiz 11.1

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Consider the system with the mathematical model given by the differential equation:

$$5 \frac{d^3 y(t)}{dt^3} + 10 \frac{d^2 y(t)}{dt^2} + 5 \frac{dy(t)}{dt} + 2y(t) = u(t)$$

Obtain a state variable model of this system.

# Quiz 11.2

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The state-space model of a system is:

$$\dot{\mathbf{x}}(t) = \begin{bmatrix} 0 & 6 \\ -1 & -5 \end{bmatrix} \mathbf{x}(t) + \begin{bmatrix} 0 \\ 1 \end{bmatrix} u(t)$$

- (1) Find the state-transition matrix;
- (2) If initial condition is  $\mathbf{x}(0) = \begin{bmatrix} 1 \\ 2 \end{bmatrix}$ , find the time response of the state variables.

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# Thank You !