



Xi'an Jiaotong-Liverpool University  
西交利物浦大學

# EEE220 Instrumentation and Control System

*2018-19 Semester 2*

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# Lecture 21

# Outline

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## Frequency Response Methods

- ☒ Introduction
- ☒ Frequency Response Plots – Polar Plot & Bode Plot
- ☐ Frequency Response Measurements
- ☐ Performance Specifications in the Frequency Domain
- ☐ Gain Margin and Phase Margin
- ☐ Compensators
- ☐ Frequency Response Methods Using Matlab

# Introduction

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In this chapter, we consider the steady-state response of a system to a **sinusoidal** input signal.

$$r(t) = A \sin \omega t$$

$$R(s) = \frac{A\omega}{s^2 + \omega^2}$$

- The frequency response of a system is defined as the steady-state response of the system to a sinusoidal input signal.
- The sinusoid is a unique input signal, and the resulting output signal for a linear constant coefficient system is sinusoidal in the steady state;
- The output signal differs from the input only in **amplitude** and **phase angle**.

# Frequency Response

Consider the system  $Y(s) = T(s)R(s)$ , with  $r(t) = A \sin \omega t$ . We have

$$R(s) = \frac{A\omega}{s^2 + \omega^2} \quad T(s) = \frac{m(s)}{q(s)} = \frac{m(s)}{\prod_{i=1}^n (s + p_i)}$$

where  $-p_i$  are assumed to be distinct poles. Then in partial forms, we have

$$Y(s) = \frac{k_1}{s + p_1} + \dots + \frac{k_n}{s + p_n} + \frac{\alpha s + \beta}{s^2 + \omega^2}$$

Taking the inverse Laplace transform yields

$$y(t) = k_1 e^{-p_1 t} + \dots + k_n e^{-p_n t} + \mathcal{L}^{-1}\left\{\frac{\alpha s + \beta}{s^2 + \omega^2}\right\}$$

where  $\alpha$  and  $\beta$  are constants which are problem dependent. If the system is stable, then all  $p_i$  should have positive real parts, the steady-state output is

$$\lim_{t \rightarrow \infty} y(t) = \mathcal{L}^{-1}\left\{\frac{\alpha s + \beta}{s^2 + \omega^2}\right\}$$

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So, in the limit for  $y(t)$ , it can be shown, for  $t \rightarrow \infty$  (the steady state),

$$\begin{aligned} y(t) &= \frac{1}{\omega} |A\omega T(j\omega)| \sin(\omega t + \phi) \\ &= A|T(j\omega)| \sin(\omega t + \phi) \end{aligned}$$

where  $\phi = \angle T(j\omega)$ .

The steady-state response described above is true only for stable systems.

Therefore, the steady-state output signal depends only on the magnitude ( $|T(j\omega)|$ ) and phase ( $\phi$ ) of  $T(j\omega)$ .

**Q: Why NOT use the final value theorem to find the steady-state output?**

# Laplace Transform vs. Fourier Transform

## Laplace transform pair

$$F(s) = \mathcal{L}\{f(t)\} = \int_0^{\infty} f(t)e^{-st} dt$$

$$f(t) = \mathcal{L}^{-1}\{F(s)\} = \frac{1}{2\pi j} \int_{\sigma-j\infty}^{\sigma+j\infty} F(s)e^{st} ds,$$

- The Laplace transform enables us to investigate the s-plane location of the poles and zeros of a transfer function  $T(s)$ .

$$s = j\omega$$



## Fourier transform pair

$$F(\omega) = \mathcal{F}\{f(t)\} = \int_{-\infty}^{\infty} f(t)e^{-j\omega t} dt$$

$$f(t) = \mathcal{F}^{-1}\{F(\omega)\} = \frac{1}{2\pi} \int_{-\infty}^{\infty} F(\omega)e^{j\omega t} d\omega.$$

- The Fourier transform allows us to consider the frequency response including amplitude and phase characteristics of the system.

# Advantages & Disadvantages of Frequency Response Method

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## Advantages:

- The ready availability of sinusoid test signals for various range of frequency and amplitudes. Thus, the experimental determination of the system frequency response is easily accomplished;
- The unknown transfer function of a system can often be deduced from the experimentally determined frequency response of the system;
- The design of a system in the frequency domain provides the designer with control of the bandwidth of a system, as well as some measure of the response of the system to undesired noise and disturbances.
- The transfer function describing the sinusoidal steady-state behavior of a system can be easily obtained by replacing  $s$  with  $j\omega$  in the system transfer function  $T(s)$ .

## Basic Disadvantage:

- Indirect link between the frequency and time domain.



# Frequency Response Plots – Polar Plot

The transfer function of a system  $G(s)$  can be described in the frequency domain by the relation

$$G(j\omega) = G(s)|_{s=j\omega} = R(\omega) + jX(\omega),$$

where

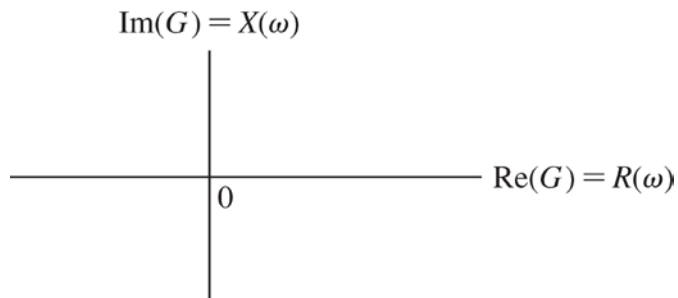
$$R(\omega) = \text{Re}[G(j\omega)] \quad \text{and} \quad X(\omega) = \text{Im}[G(j\omega)].$$

Alternatively, the transfer function can be represented by magnitude and phase as

$$G(j\omega) = |G(j\omega)|e^{j\phi(\omega)} = |G(j\omega)|\angle\phi(\omega),$$

where

$$\phi(\omega) = \tan^{-1} \frac{X(\omega)}{R(\omega)} \quad \text{and} \quad |G(j\omega)|^2 = [R(\omega)]^2 + [X(\omega)]^2.$$



The **polar plot** representation of the frequency response is obtained by using the above equations.

# Example 21.1: Frequency Response of an RC Filter

A simple RC filter is shown in the right figure.  
The transfer function is

$$G(s) = \frac{V_2(s)}{V_1(s)} = \frac{1}{RCs + 1}$$

and the sinusoidal steady-state transfer function is

$$G(j\omega) = \frac{1}{j\omega(RC) + 1} = \frac{1}{j(\omega/\omega_1) + 1} \quad \text{where } \omega_1 = \frac{1}{RC}$$

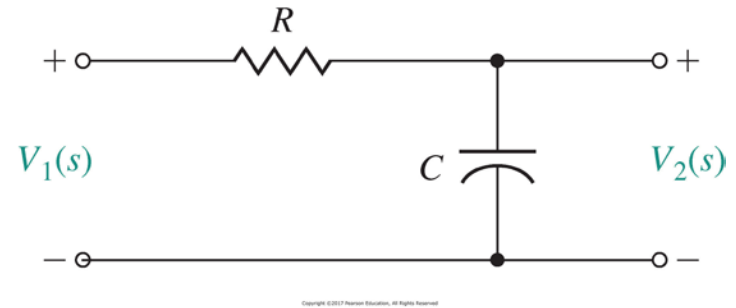
Then the polar plot is obtained from the relation

$$G(j\omega) = \frac{1}{1 + (\frac{\omega}{\omega_1})^2} - j \frac{\frac{\omega}{\omega_1}}{1 + (\frac{\omega}{\omega_1})^2}$$

or

$$G(j\omega) = |G(j\omega)| \angle \phi(\omega),$$

where  $|G(j\omega)| = [\frac{1}{1 + (\frac{\omega}{\omega_1})^2}]^{1/2}$  and  $\phi(\omega) = -\tan^{-1}(\frac{\omega}{\omega_1})$

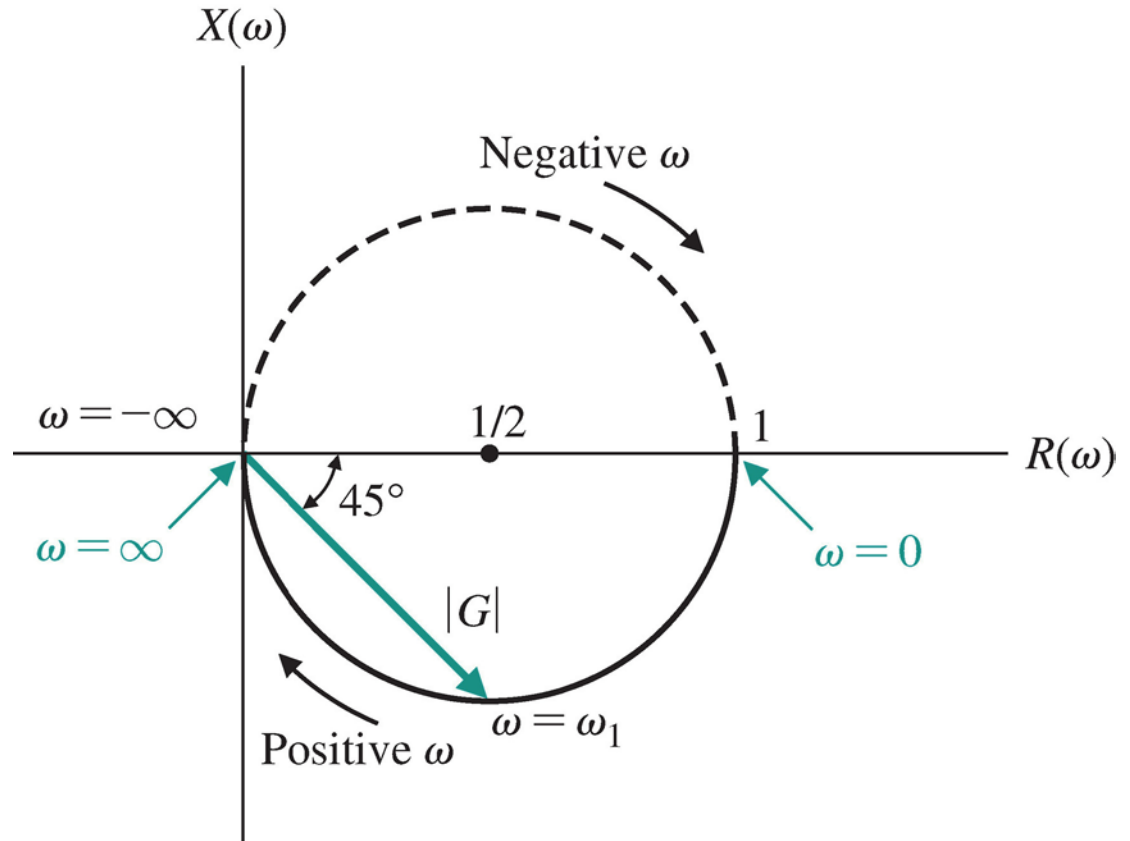


# Polar Plot for the RC Filter

$$G(j\omega) = |G(j\omega)| \angle \phi(\omega),$$

$$\text{where } |G(j\omega)| = \left[ \frac{1}{1 + \left(\frac{\omega}{\omega_1}\right)^2} \right]^{1/2}$$

$$\text{and } \phi(\omega) = -\tan^{-1}\left(\frac{\omega}{\omega_1}\right)$$



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# Example 21.2: Polar Plot of a Transfer Function

Consider a transfer function

$$G(j\omega) = \frac{K}{j\omega(j\omega\tau + 1)} = \frac{K}{j\omega - \omega^2\tau}$$

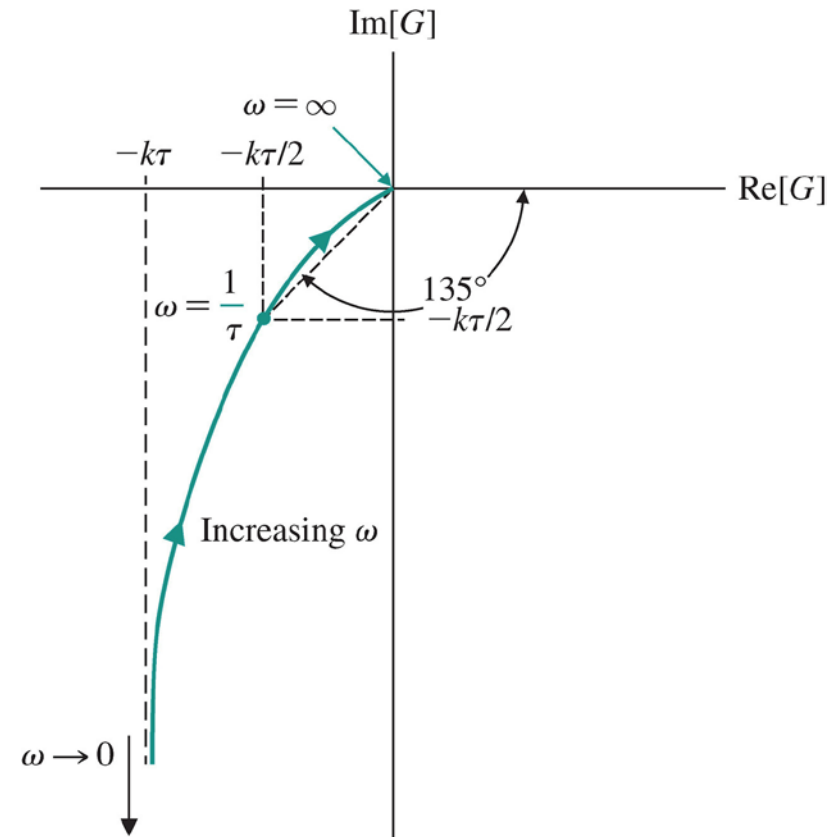
Then the magnitude and phase angle are written as

$$|G(j\omega)| = \frac{K}{(\omega^2 + \omega^4\tau^2)^{1/2}}$$

$$\phi(\omega) = -\tan^{-1} \frac{1}{-\omega\tau}.$$

Typical values:

$\omega$	0	$1/2\tau$	$1/\tau$	$\infty$
$ G(j\omega) $	$\infty$	$4K\tau/\sqrt{5}$	$K\tau/\sqrt{2}$	0
$\phi(\omega)$	$-90^\circ$	$-117^\circ$	$-135^\circ$	$-180^\circ$



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# Bode Plots

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The introduction of **logarithm plots**, often called **Bode Plots** in honor of H. W. Bode, simplifies the determination of the graphical portrayal of the frequency response.

The transfer function in the frequency domain is

$$G(j\omega) = |G(j\omega)|e^{j\phi(\omega)}$$

The logarithm of the magnitude is normally expressed in terms of the logarithm to the **base 10**, so we use

$$\text{Logarithm Gain} = 20\log_{10}|G(j\omega)|$$

where the units are **decibels (dB)**.

For a Bode diagram, the plot of logarithmic gain in dB versus  $\omega$  is normally plotted on one set of axes, and the phase  $\phi(\omega)$  versus  $\omega$  on another set of axes.

# Bode Plot of an RC Filter

Reconsider the transfer function

$$G(j\omega) = \frac{1}{j\omega(RC) + 1} = \frac{1}{j\omega\tau + 1} \quad \text{where } \tau = RC$$

The logarithm gain is

$$20 \log|G(j\omega)| = 20 \log\left(\frac{1}{1 + (\omega\tau)^2}\right)^{1/2} = -10 \log(1 + (\omega\tau)^2).$$

For small frequencies – that is,  $\omega \ll 1/\tau$ , the logarithm gain is

$$20 \log|G(j\omega)| = -10 \log(1) = 0 \text{ dB}, \quad \omega \ll 1/\tau.$$

For large frequencies – that is,  $\omega \gg 1/\tau$ , the logarithm gain is

$$20 \log|G(j\omega)| = -20 \log(\omega\tau) \quad \omega \gg 1/\tau,$$

and at  $\omega = 1/\tau$  (**break frequency** or **corner frequency**), we have

$$20 \log|G(j\omega)| = -10 \log 2 = -3.01 \text{ dB}.$$

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Phase angle of the transfer function is

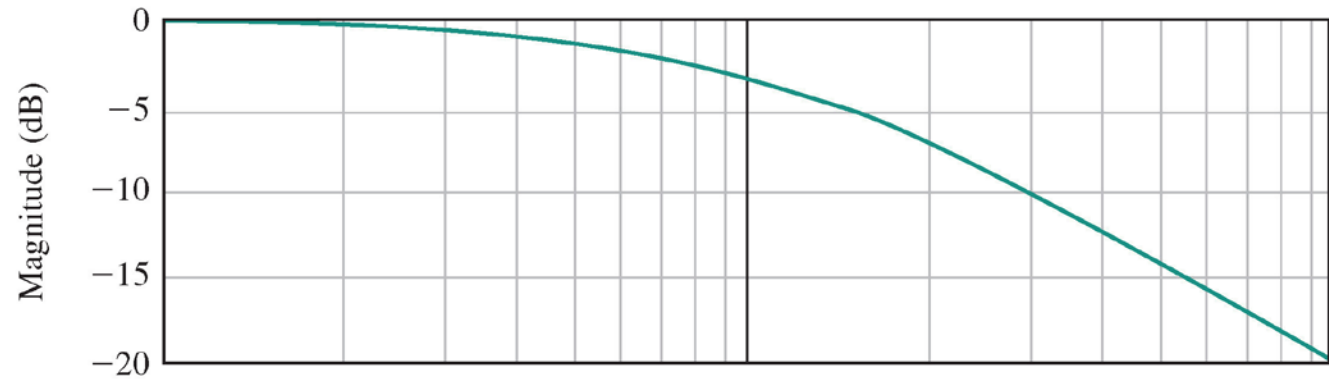
$$\phi(\omega) = -\tan^{-1}(\omega\tau).$$

- A linear scale of frequency is not the most convenient choice, we consider the use of a logarithmic scale of frequency. Then, on a set of axes where the horizontal axis is **log $\omega$** .
- An interval of two frequencies with a ratio equal to 10 is called a **decade**, so that the range of frequencies from  $\omega_1$  to  $\omega_2$ , where  $\omega_2 = 10\omega_1$ , is called a decade.
- The logarithmic gains, for  $\omega \gg 1/\tau$ , over a decade of frequency is

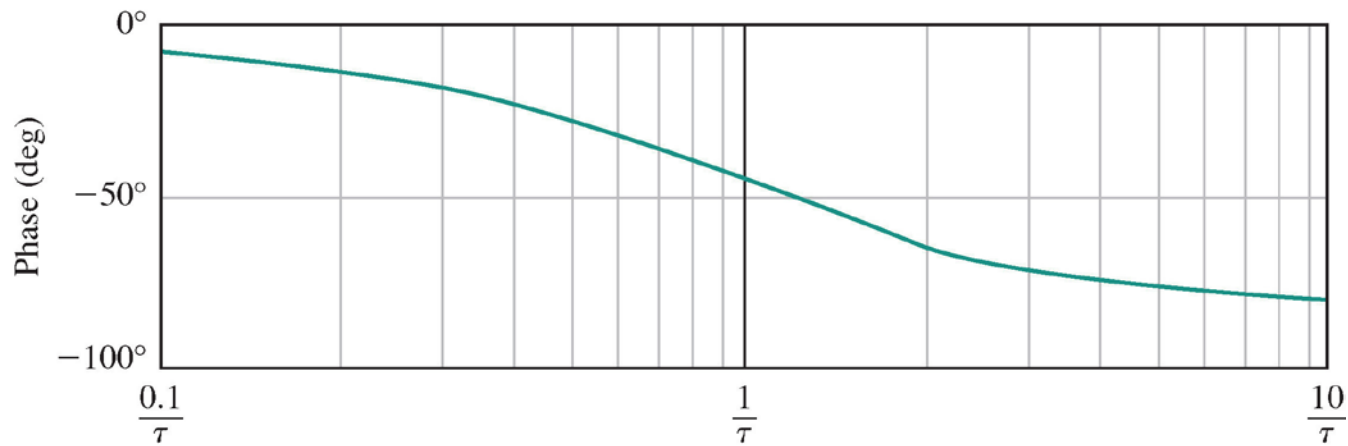
$$\begin{aligned} 20 \log|G(j\omega_1)| - 20 \log|G(j\omega_2)| &= -20 \log(\omega_1\tau) - (-20 \log(\omega_2\tau)) \\ &= -20 \log \frac{\omega_1\tau}{\omega_2\tau} \\ &= -20 \log \frac{1}{10} = +20 \text{ dB}; \end{aligned}$$

That is, the slope of the asymptotic line for this first-order transfer function is -20 dB/decade.

Bode plot for  $G(j\omega) = 1/(j\omega\tau + 1)$ ; (a) magnitude plot and (b) phase plot.



(a)



Frequency (rad/s)

(b)

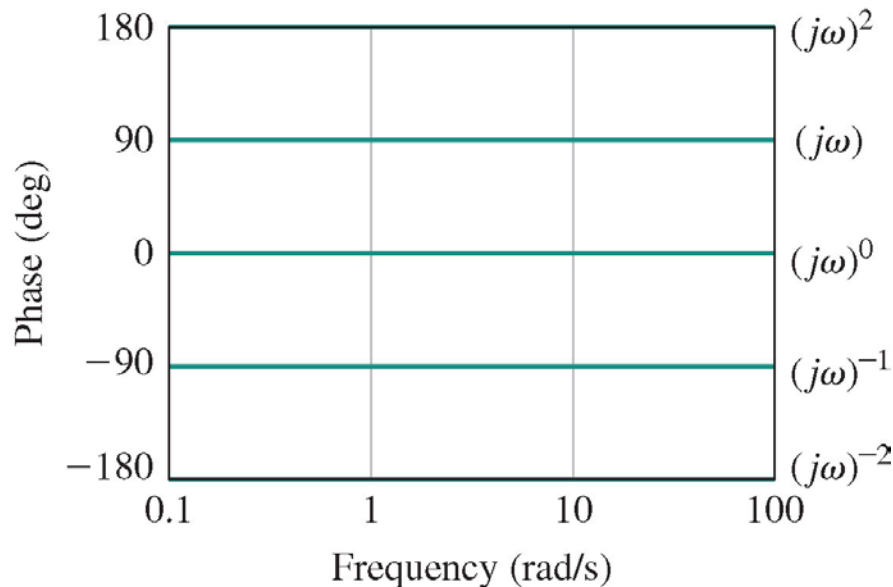
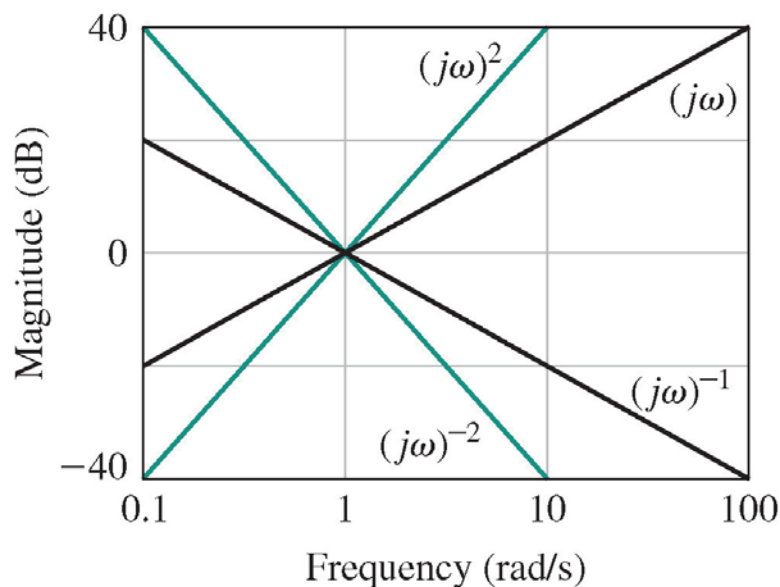


# Bode Plots for Typical Transfer Functions

Bode plot for  $(j\omega)^{\pm N}$ . -- Poles (or Zeros) at the Origin.

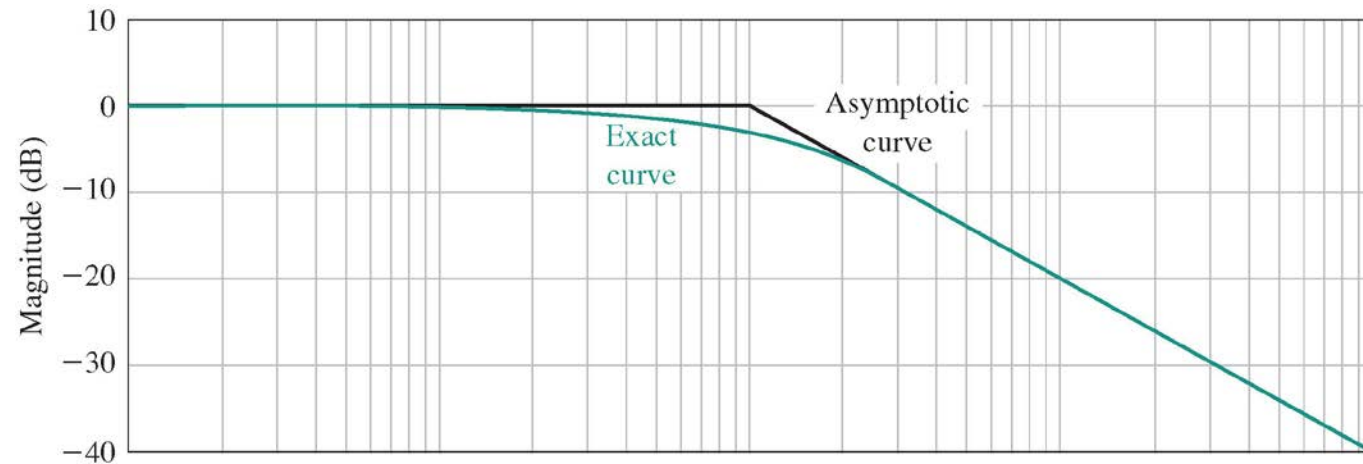
$$20 \log \left| \frac{1}{(j\omega)^N} \right| = \pm 20N \log \omega,$$

$$\phi(\omega) = \pm 90^\circ N.$$

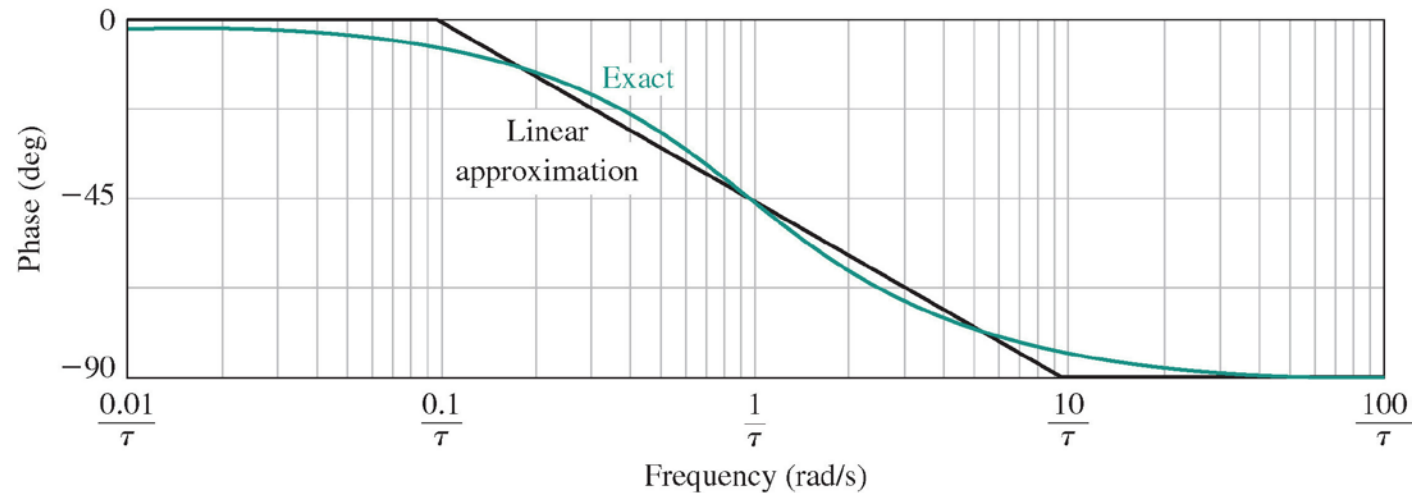


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Bode diagram for  $(1 + j\omega\tau)^{-1}$ . -- Poles (or Zeros) on the Real Axis.



(a)



(b)

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Bode diagram for  $G(j\omega) = [1 + (2\zeta/\omega_n) j\omega + (j\omega/\omega_n)^2]^{-1}$ .

-- Complex Conjugate Poles or Zeros.

Normalized form  $[1 + j2\zeta u - u^2]^{-1}$

where  $u = \omega/\omega_n$ .

$$20 \log|G(j\omega)| = -10 \log((1 - u^2)^2 + 4\zeta^2 u^2),$$

$$\phi(\omega) = -\tan^{-1} \frac{2\zeta u}{1 - u^2}.$$

when  $u \ll 1$

$$20 \log|G(j\omega)| = -10 \log 1 = 0 \text{ dB},$$

While the phase angle approaches  $0^\circ$

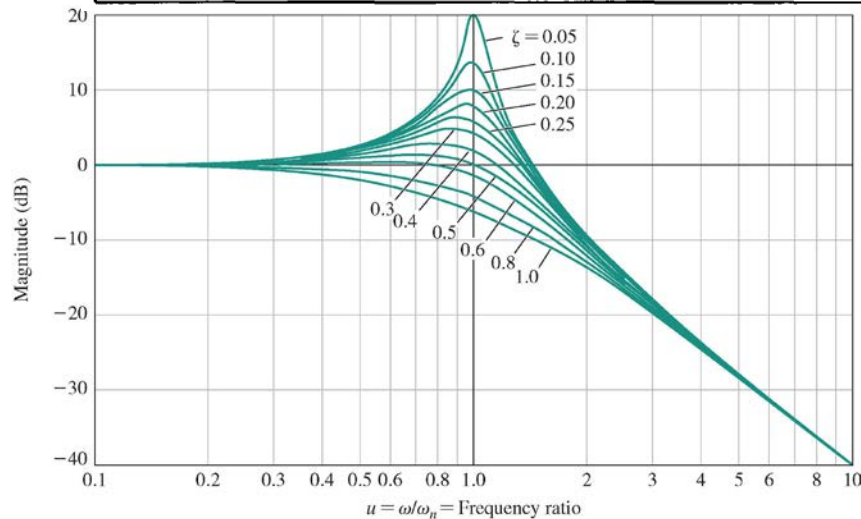
when  $u \gg 1$

$$20 \log|G(j\omega)| = -10 \log u^4 = -40 \log u,$$

While the phase angle approaches  $-180^\circ$

The difference between the actual magnitude curve and the asymptotic approximation is a function of the **damping ratio** and **MUST be accounted for when  $\zeta < 0.707$** .

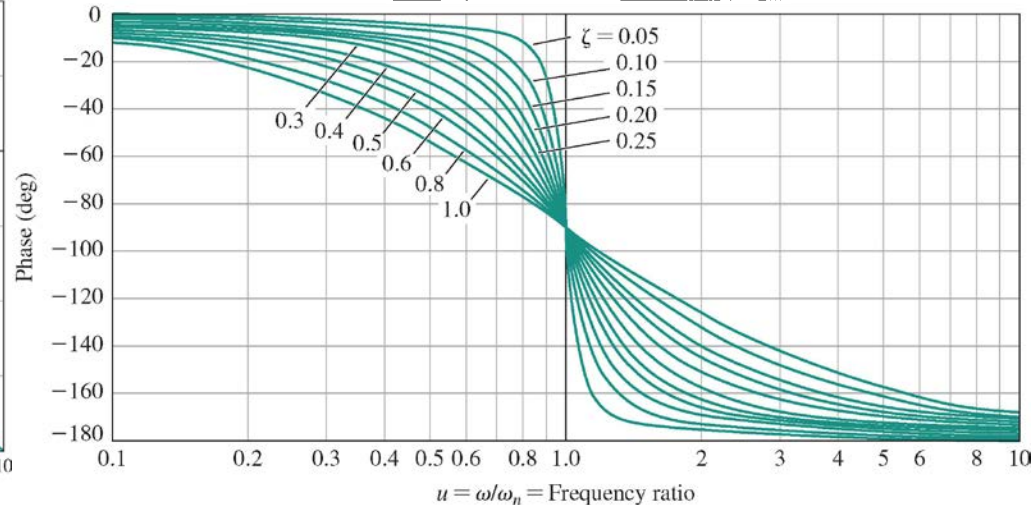
$$20 \log|G(j\omega)| = -10 \log((1 - u^2)^2 + 4\zeta^2 u^2),$$



(a)

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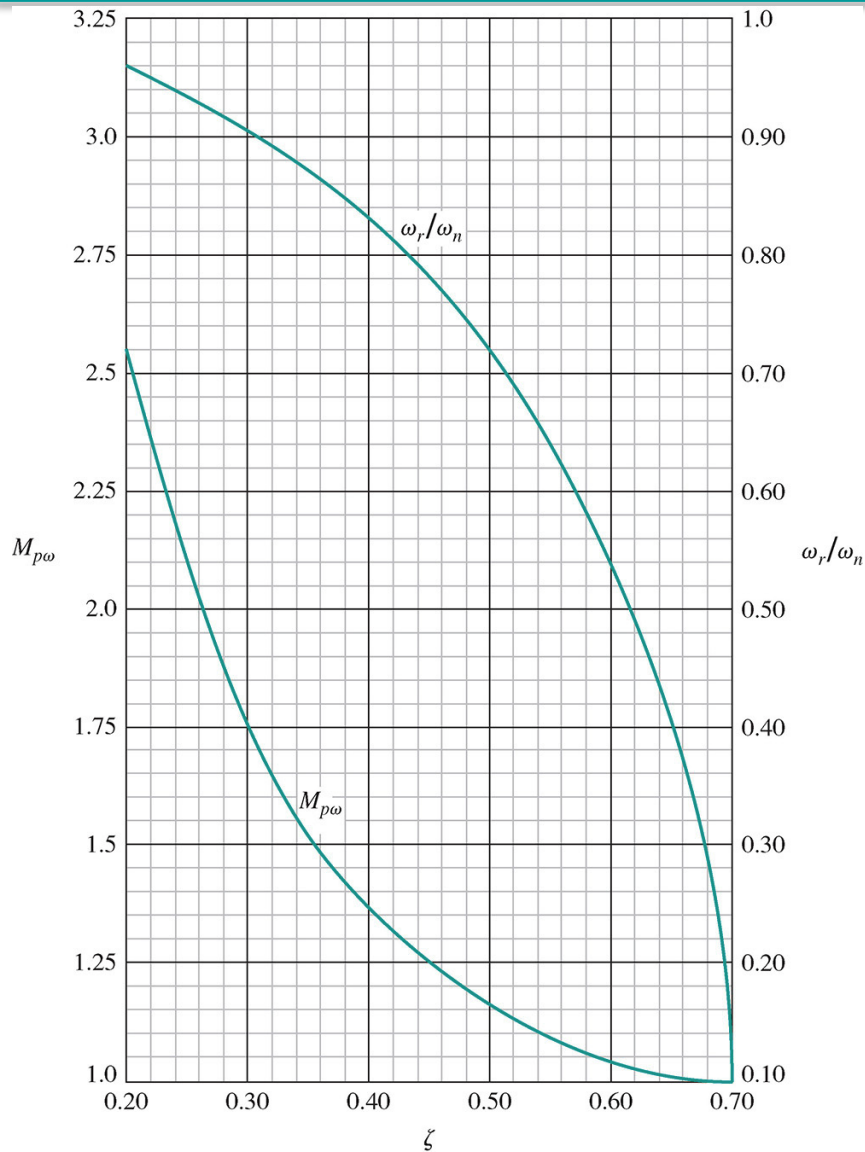
$$\phi(\omega) = -\tan^{-1} \frac{2\zeta u}{1 - u^2}.$$



(b)

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The maximum value  $M_{p\omega}$  of the frequency response occurs at the resonant frequency  $\omega_r$ .  
When the damping ratio approaches zero, then  $\omega_r$  approaches  $\omega_n$ .

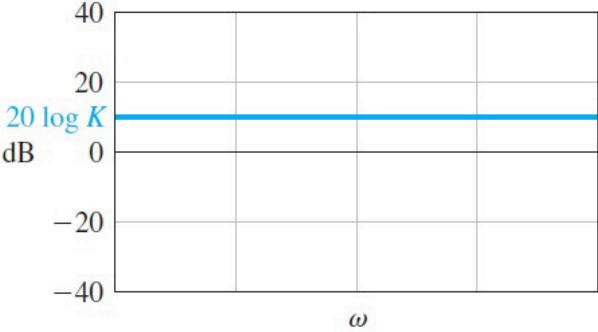
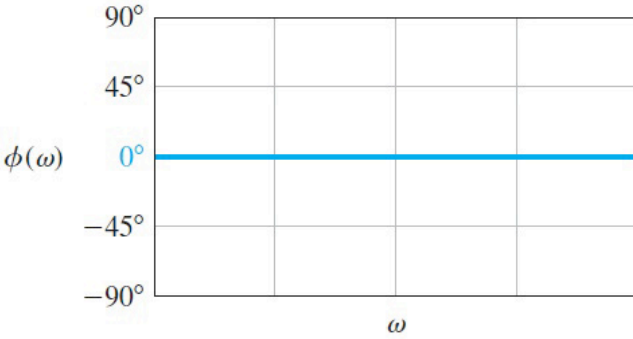
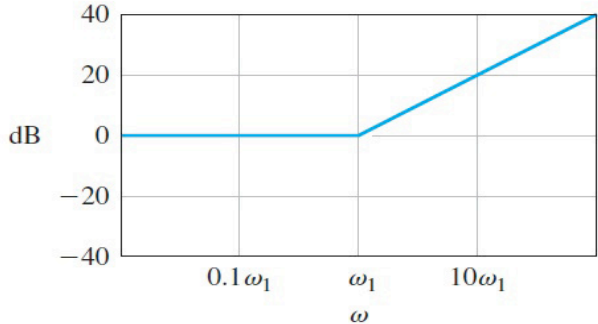
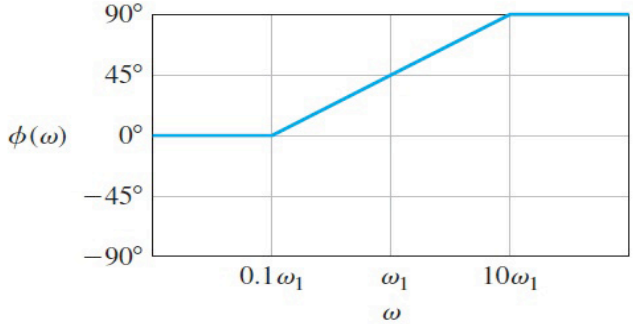
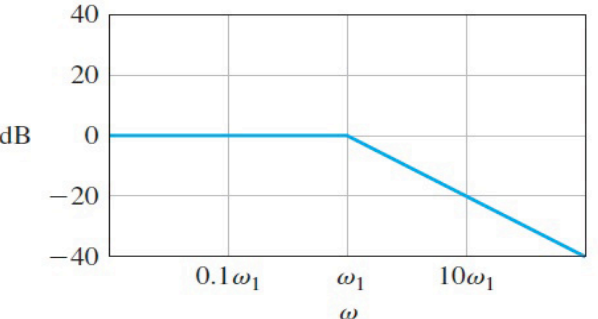
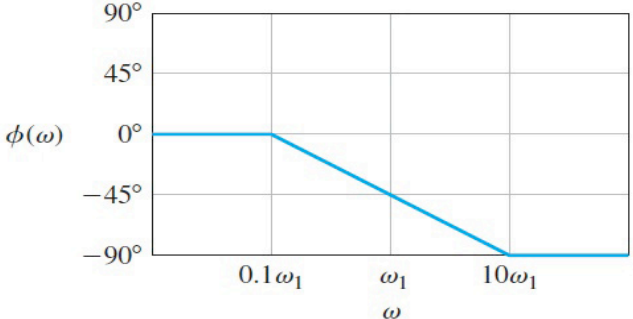


$$\omega_r = \omega_n \sqrt{1 - 2\zeta^2}, \quad \zeta < 0.707.$$

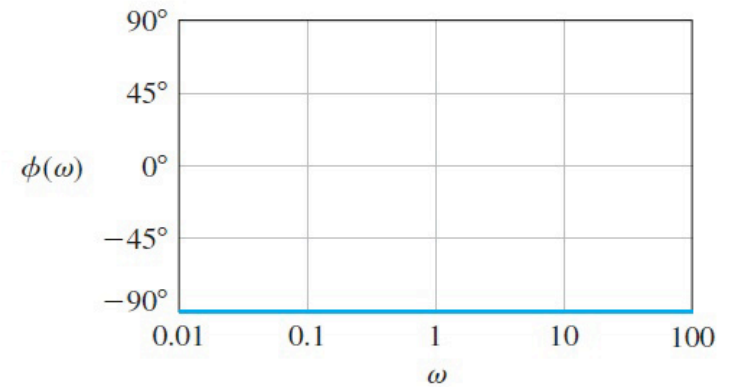
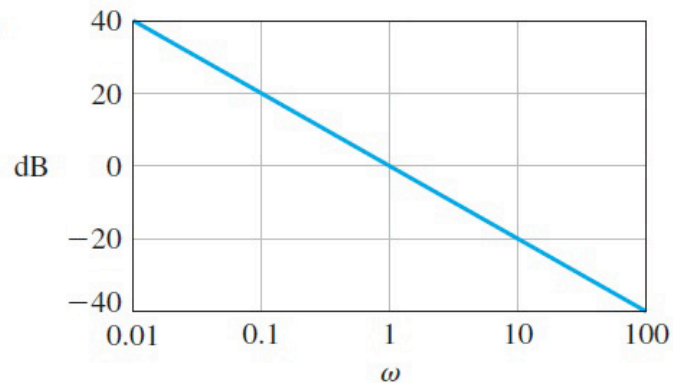
$$M_{p\omega} = |G(j\omega_r)| = (2\zeta\sqrt{1 - \zeta^2})^{-1}, \quad \zeta < 0.707.$$

The maximum  $M_{p\omega}$  of the frequency response and the resonant frequency  $\omega_r$  versus  $\zeta$  for a pair of conjugate poles.

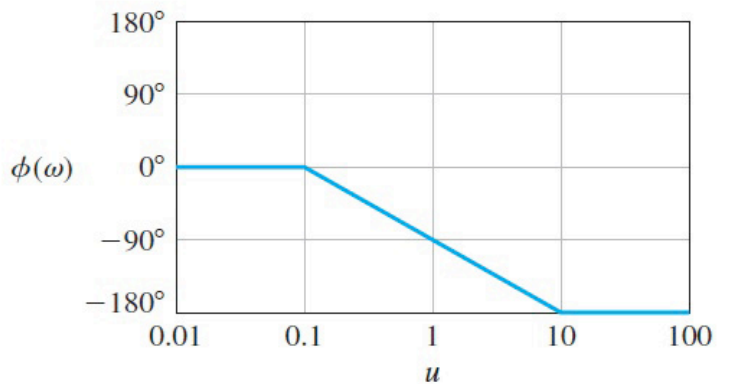
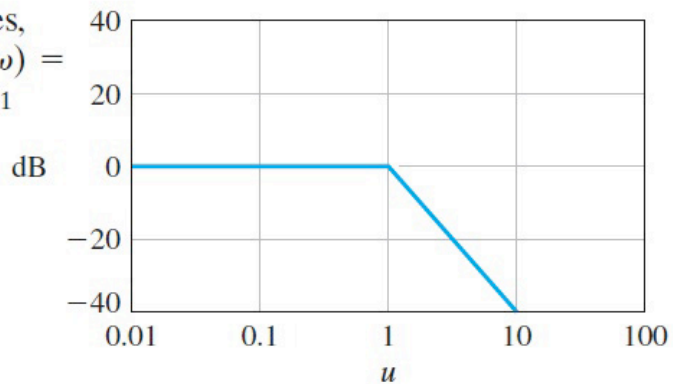
# Summary: Asymptotic Curves for Basic Terms of a Transfer Function

Term	Magnitude $20 \log G $	Phase $\phi(\omega)$
1. Gain, $G(j\omega) = K$		
2. Zero, $G(j\omega) = \frac{1}{1 + j\omega/\omega_1}$		
3. Pole, $G(j\omega) = (1 + j\omega/\omega_1)^{-1}$		

4. Pole at the origin,  
 $G(j\omega) = 1/j\omega$



5. Two complex poles,  
 $0.1 < \zeta < 1$ ,  $G(j\omega) =$   
 $(1 + j2\zeta u - u^2)^{-1}$   
 $u = \omega/\omega_n$



# Sketching a Bode Plot

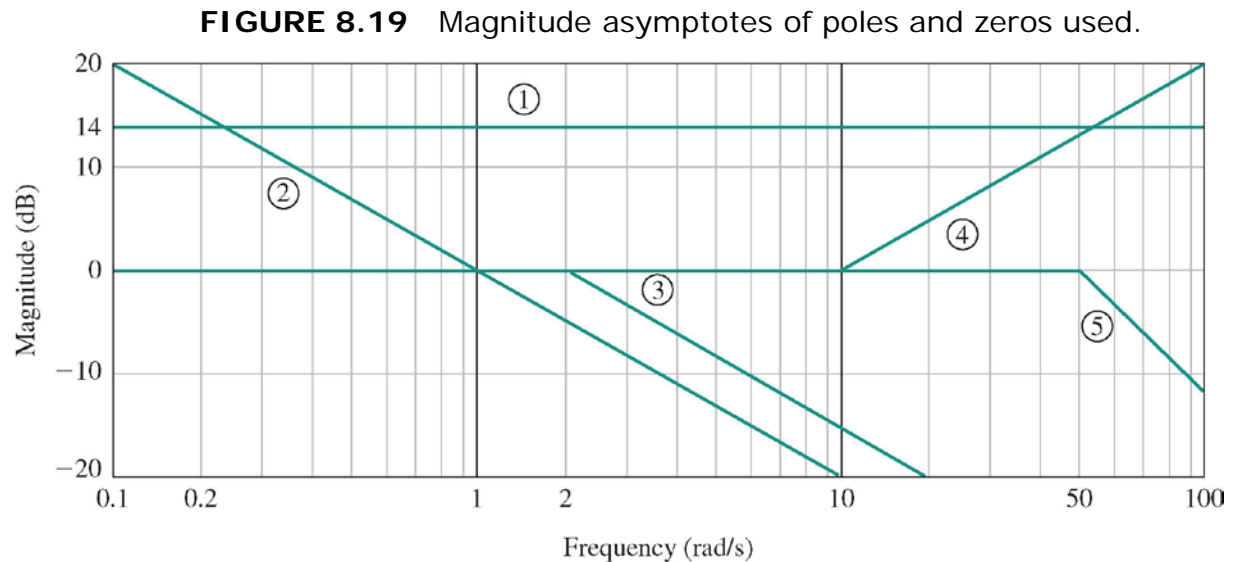
Consider the system with the following transfer function

$$G(j\omega) = \frac{5(1 + j0.1\omega)}{j\omega(1 + j0.5\omega)(1 + j0.6(\omega/50) + (j\omega/50)^2)}$$

## Solutions:

There are five terms in the transfer function:

1. A constant gain  $K = 5$
2. A pole at the origin
3. A pole at  $\omega = 2$
4. A zero at  $\omega = 10$
5. A pair of complex poles at  $\omega = \omega_n = 50$



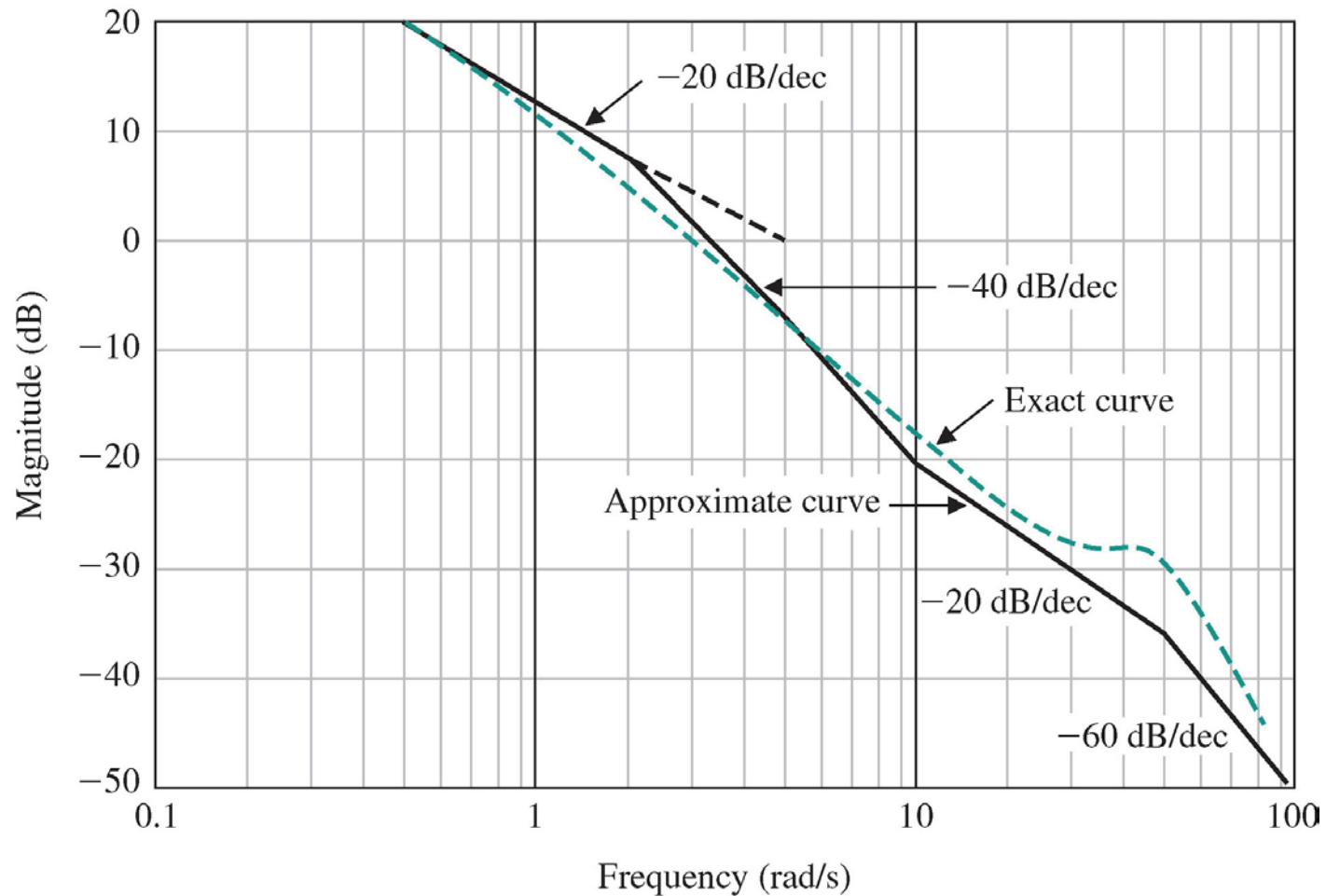


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First, we plot the magnitude characteristic for each individual pole and zero factor and the constant gain:

1. The constant gain is  $20 \log 5 = 14$  dB, as shown in Figure 8.19.
2. The magnitude of the pole at the origin extends from zero frequency to infinite frequencies and has a slope of  $-20$  dB/decade intersecting the 0-dB line at  $\omega = 1$ , as shown in Figure 8.19.
3. The asymptotic approximation of the magnitude of the pole at  $\omega = 2$  has a slope of  $-20$  dB/decade beyond the break frequency at  $\omega = 2$ . The asymptotic magnitude below the break frequency is 0 dB, as shown in Figure 8.19.
4. The asymptotic magnitude for the zero at  $\omega = +10$  has a slope of  $+20$  dB/decade beyond the break frequency at  $\omega = 10$ , as shown in Figure 8.19.
5. The magnitude for the complex poles is  $-40$  dB/decade. The break frequency is  $\omega = \omega_n = 50$ , as shown in Figure 8.19. This approximation must be corrected to the actual magnitude because the damping ratio is  $\zeta = 0.3$ , and the magnitude differs appreciably from the approximation, as shown in Figure 8.20.

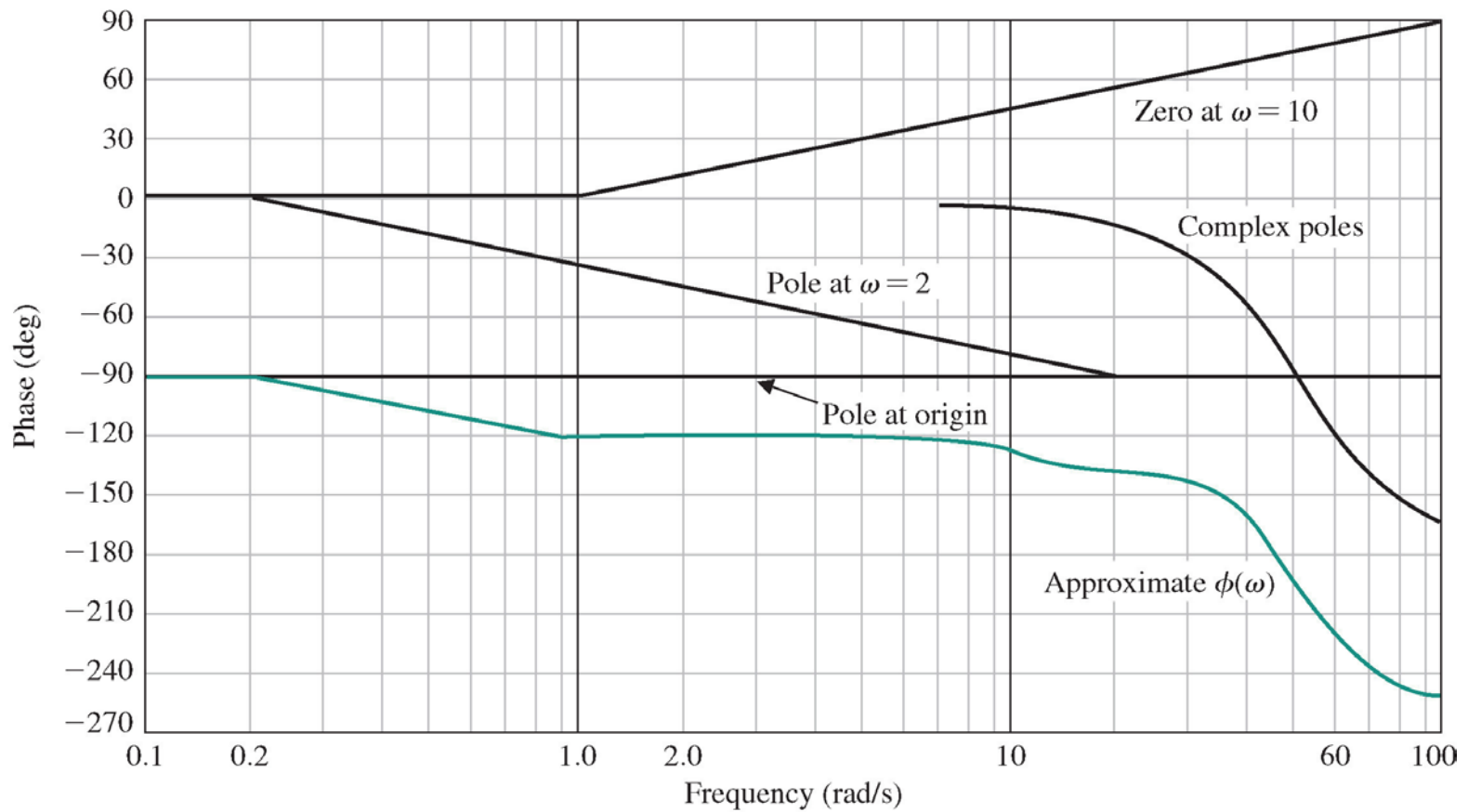
**FIGURE 8.20** Magnitude characteristic.



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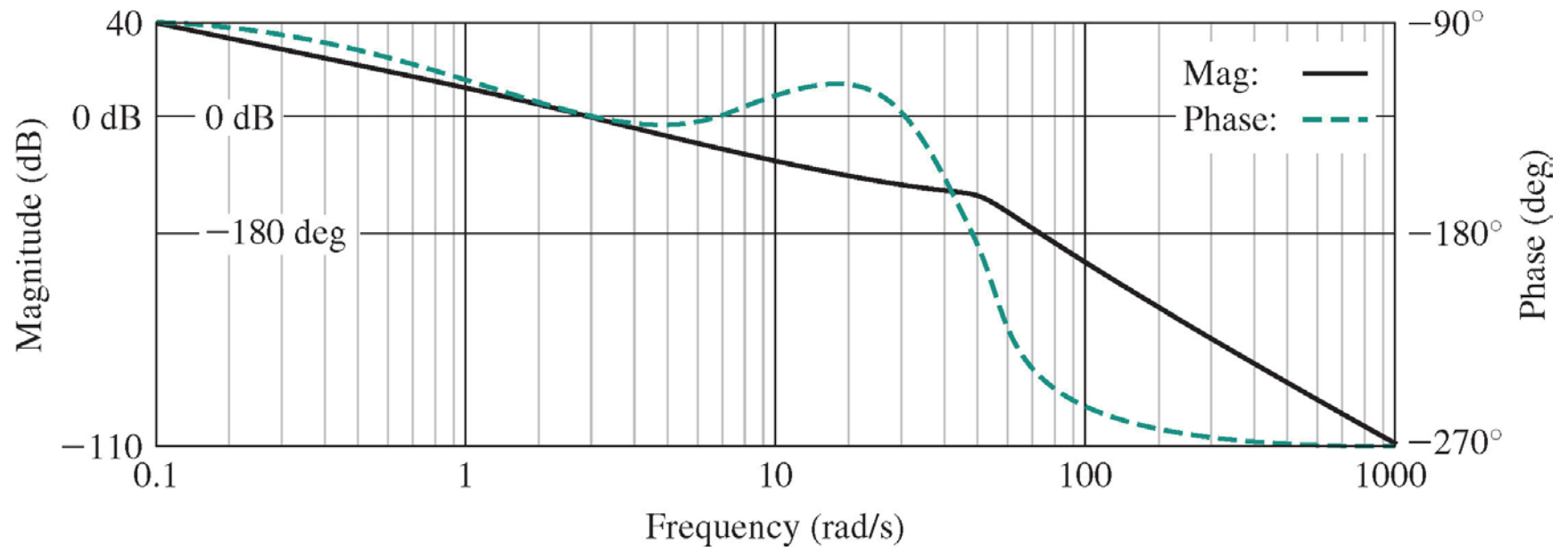
- 
1. The phase of the constant gain is  $0^\circ$ .
  2. The phase of the pole at the origin is a constant  $-90^\circ$ .
  3. The linear approximation of the phase characteristic for the pole at  $\omega = 2$  is shown in Figure 8.21, where the phase shift is  $-45^\circ$  at  $\omega = 2$ .
  4. The linear approximation of the phase characteristic for the zero at  $\omega = 10$  is also shown in Figure 8.21, where the phase shift is  $+45^\circ$  at  $\omega = 10$ .
  5. The actual phase characteristic for the pair of complex poles is obtained from Figure 8.10 and is shown in Figure 8.21.

**FIGURE 8.21** Phase characteristic.



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## The Bode plot of the $G(j\omega)$



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# Quiz 21.1

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For the following transfer function

$$L(s) = G_c(s)G(s) = \frac{300(s + 100)}{s(s + 10)(s + 40)}.$$

1. determine the phase angle  $\phi(\omega)$  when  $\omega = 28.3$  rad/s;
2. Find the logarithmic gain of  $L(j\omega)$  at  $\omega = 28.3$  rad/s.
3. Sketch the Bode plots for this transfer function.

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# Thank You !