

# HW4

December 1, 2025

## 1 HOMEWORK4

Consider a single-degree-of-freedom spring-mass-damper system with mass  $m = 3\text{kg}$ , undamped natural circular frequency  $\omega_n = 16\text{rad/s}$  and damping ratio  $\zeta = 0.06$ . The system is subjected to an excitation force  $f(t)$  that consists of a sequence of two half-sine pulses, as shown in the figure below.

The first pulse has a height magnitude of 5 N and a frequency  $\omega_1 = 13\text{rad/s}$ , and starts from zero at time  $t = 0$  seconds. The second pulse has a height magnitude of 11 N and a frequency of  $\omega_2 = 20\text{rad/s}$ . The second pulse starts from zero at  $t = T1$ , where  $T1 = \frac{2\pi}{\omega_1}$  is the period of a full oscillation at frequency  $\omega_1$ .

Convolutional integral:

$$x(t) = \int_0^t f(\tau)g(t - \tau)d\tau$$

Where  $f(t)$  is excitation. From equation 4.13, impulse response is:

$$g(t) = \frac{1}{m\omega_d} e^{-\zeta\omega_n t} \sin\omega_d t$$

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[87]: #from numpy import sqrt, exp
import sympy
from sympy import symbols, exp, sin, sqrt, integrate
from numpy import vectorize, linspace, pi
import matplotlib.pyplot as plt

# system parameters
m = 3
w_n = 16
zeta = 0.06
w_d = w_n * sqrt(1 - zeta**2)
t, tau = symbols('t tau', real=True, positive=True)
g = 1/(m*w_d) * exp(-zeta*w_n*t)*sin(w_d*t)
g_tau = (1 / (m * w_d)) * exp(-zeta * w_n * (t - tau)) * sin(w_d * (t - tau))
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[88]: F_1 = 5
F_2 = 11
w_1 = 13
w_2 = 20
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T_1 = 2*pi/w_1
T_2 = 2*pi/w_2
T = [0.0, 5 * T_1]
t_vals = linspace(0, 5*T_1, 2000)

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[89]: #impluse excitation
f_1 = -F_1 * sin(w_1*tau)
f_2 = F_2 * sin(w_2 * (tau-T_1))

def f_t(t):
    if 0 <= t <= T_1/2:
        return -F_1 * sin(w_1 * t)
    elif T_1 <= t <= T_1 + T_2/2:
        return F_2 * sin(w_2 * (t-T_1))
    else:
        return 0

f_vec = vectorize(f_t)

def solve_convolution(f_func, lower_limit, upper_limit):
    integrand = f_func * g_tau
    result = integrate(integrand, (tau, lower_limit, upper_limit))
    return result

x_1 = solve_convolution(f_1, 0, t) # 0 < t < T_1/2
x_2 = solve_convolution(f_1, 0, T_1/2) # T_1/2 < t < T_1
x_3 = solve_convolution(f_2, T_1, t) # T_1 < t < T_1 + T_2/2
x_4 = solve_convolution(f_2, T_1, T_1 + T_2/2) #T_1 + T_2/2 < t < T_2
x_total = sympy.Piecewise(
    (x_1, (t >= 0) & (t <= T_1/2)), # pulse 1
    (x_2, (t > T_1/2) & (t < T_1)), # free vibration
    (x_2 + x_3, (t >= T_1) & (t <= T_1 + T_2/2)), # pulse 2
    (x_2 + x_4, True) # final free vibration
)

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[90]: # Force plot
plt.figure(figsize=(10,4))
plt.plot(t_vals, f_vec(t_vals))
plt.title("Excitation Force f(t)")
plt.xlabel("Time (s)")
plt.ylabel("Force (N)")
plt.grid()

# Response plot
x_total_func = sympy.lambdify(t, x_total, 'numpy')
x_vals = x_total_func(t_vals) #      t_vals

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plt.figure(figsize=(10,4))
plt.plot(t_vals, x_vals) # t_vals x_vals
plt.title("System Response x(t)")
plt.xlabel("Time (s)")
plt.ylabel("Displacement (m)")
plt.grid()

plt.show()

```

