

Climate Modelling and Data Analysis

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Schedule

1. What are models?

Exercise: Python Skills

2. Energy budget of the Earth

Exercise: Simple Energy Balance Model

3. Nonlinearity, Feedback, and Predictability

Exercise: Nonlinearity and Feedbacks

Exercise: Revised Energy Balance Model

4. Parametrization and Sensitivity

5. Radiative budget

Exercise: 1-layer greenhouse model

Exercise: 2-layer greenhouse model

6. Introduction to fluid dynamics

Exercises: The use of the governing equations

Exercise: Analytical katabatic flow model

7. Finite difference method

Exercise: Advection-Diffusion Equation

Exercise: Boundary layer Evolution

Exercise: Numerical katabatic flow model

Exercise: Heat Equation

8. Implicit finite difference methods

Exercise: Boundary layer evolution

9. Optimization problem

Exercise: Surface energy balance

Exercise: Sublimation

10. COSIPY snow model

Exercise: Simulations with COSIPY

11. Introduction to PALM

Exercise: Simulations with PALM

12. How to write an article

Learning objective

Learning objectives

- Fundamental concepts of fluid dynamics
- Governing equations
- Discretization
- Finite-difference method

Literature

Recommended textbooks

- Young et. al, 2011: **Introduction to Fluid Mechanics**. Wiley.
- Jacobson M., 2005: **Fundamentals of atmospheric modelling**. Cambridge University Press.
- Kundu P.K., Cohen I.M., and Dowling D.R., 2016: **Fluid Mechanics**. Elsevier.
- Stull, 1988: **An Introduction to Boundary Layer Meteorology**. Springer.

Governing equations for atmospheric flow

- Conservation of mass (continuity equation)
- Conservation of momentum (Newton's second law)
- Conservation of heat (First law of thermodynamics)
- Conservation of moisture
- Conservation of s scalar quantities
- Equation of state

Basics concept of fluid dynamics

Assumptions

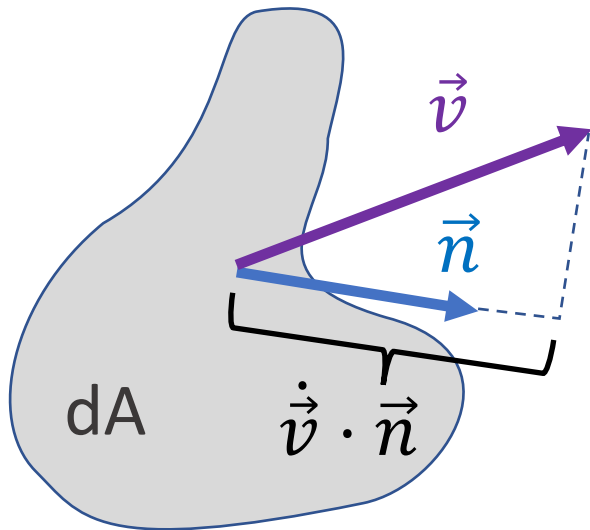
- No molecular structure (no resistance to external shear forces)
- Fluid regarded as continuum
- Flow by external forces
- Fluids are similar under action of forces

Conservation of mass (continuity equation)

Conservation law \rightarrow Basic idea

$$\left[\begin{array}{c} \text{Rate of inflow} \\ \text{to a system} \end{array} \right] - \left[\begin{array}{c} \text{Rate of outflow} \\ \text{from a system} \end{array} \right] = \left[\begin{array}{c} \text{Rate of} \\ \text{accumulation} \end{array} \right]$$

What is the rate of mass flow through a surface dA with normal vector \vec{n} ?



$\vec{v} \cdot \vec{n}$ is the projection of \vec{v} normal to the surface

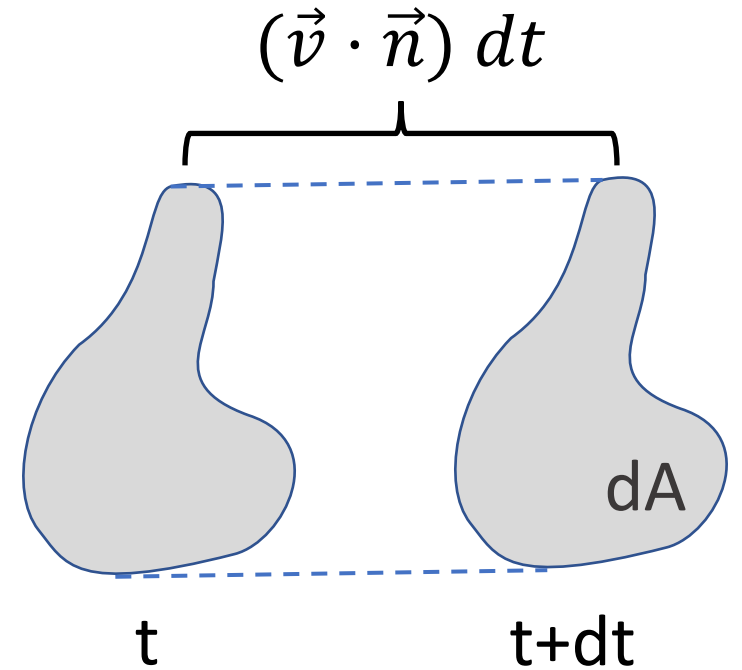
Conservation of mass

What **volume** of fluid flows through dA in time dt ?

$$(\vec{v} \cdot \vec{n}) dt dA$$

Flow through Surface Time Area

$$\left[\frac{m}{s}\right] [s] [m^2] = [m^3]$$



Conservation of mass

What **mass** of fluid flows through dA in time dt ?

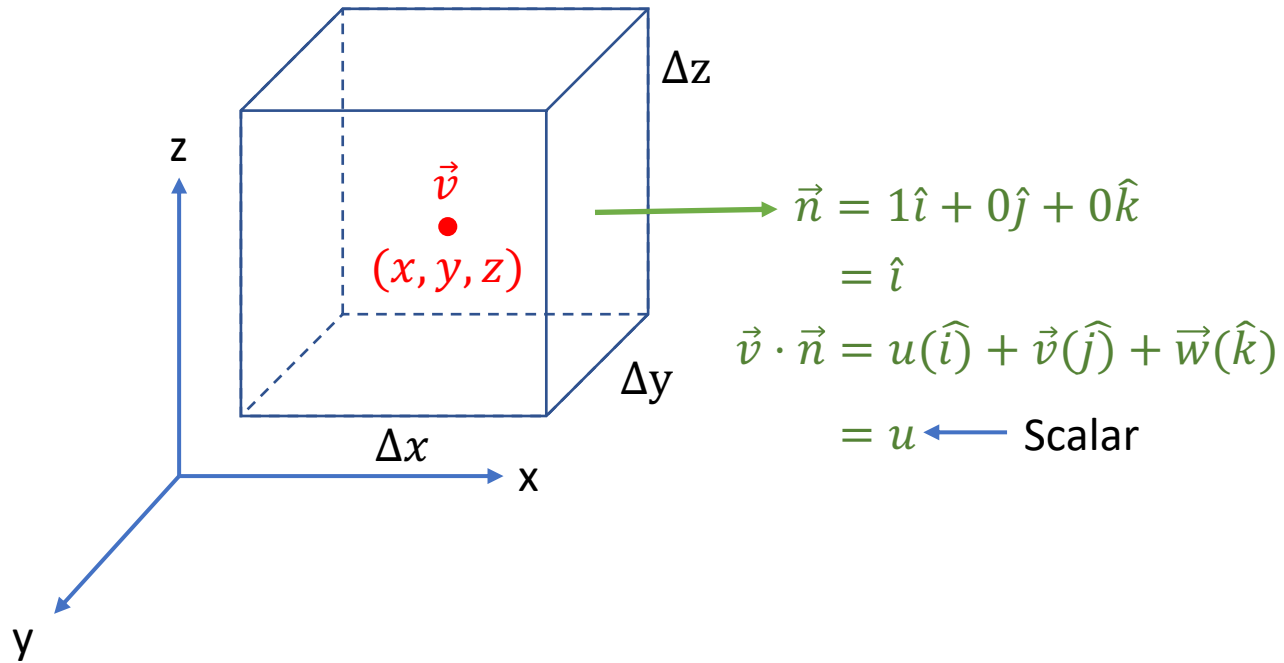
$$\text{Density } \rho = \frac{\text{mass}}{\text{volume}} \quad \longrightarrow \quad \text{mass} = (\text{density})(\text{volume}) = \rho(\vec{v} \cdot \vec{n}) dt dA$$
$$[kg] = \left[\frac{kg}{m^3} \right] [m^3]$$

$$\frac{\text{mass}}{\text{time}} = \dot{m} = \rho(\vec{v} \cdot \vec{n}) dA$$

$$\text{Mass flux (mass per unit time per unit area)} = \rho(\vec{v} \cdot \vec{n})$$

Conservation of mass

Let's write a conservation law ...



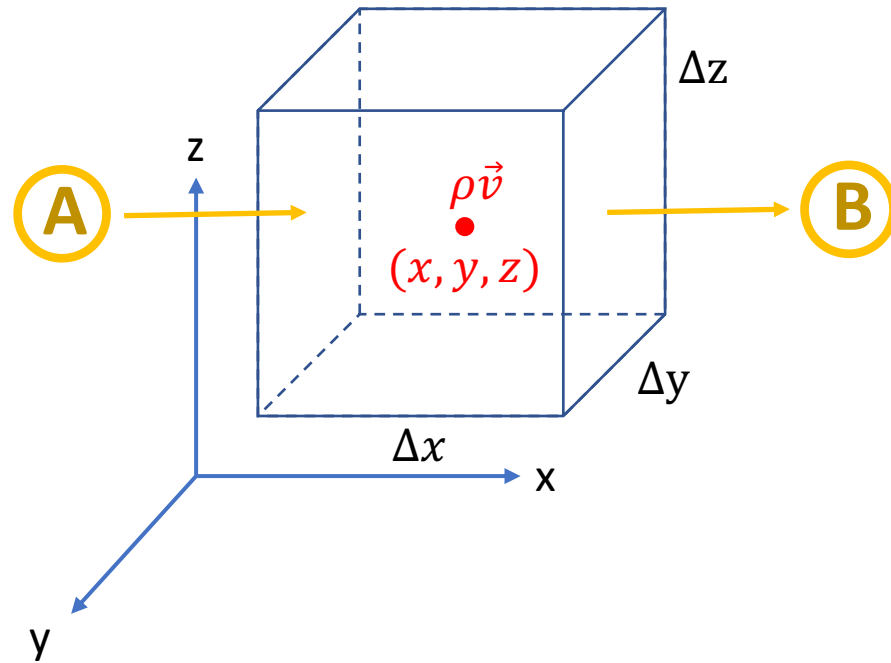
We know the following:

$$1. \vec{v} = u\hat{i} + v\hat{j} + w\hat{k}$$

$$2. \text{mass flux} = \frac{\text{mass}}{(\text{time})(\text{area})} = \rho(\vec{v} \cdot \vec{n})$$

Conservation of mass

Let's write a conservation law ...



We know the following:

$$1. \vec{v} = u\hat{i} + v\hat{j} + w\hat{k}$$

$$2. \text{mass flux} = \frac{\text{mass}}{(\text{time})(\text{area})} = \rho(\vec{v} \cdot \vec{n})$$

Mass flux IN
(x-face **A**)

$$\frac{\text{mass}}{(\text{time})(\text{area})} \text{ at center of cube } (x, y, z) \rightarrow \underbrace{\rho u + \frac{\partial(\rho u)}{\partial x} \left(-\frac{\Delta x}{2} \right)}_{\text{Change in } \rho u \text{ from center to face A}}$$

Mass flux OUT
(x-face **B**)

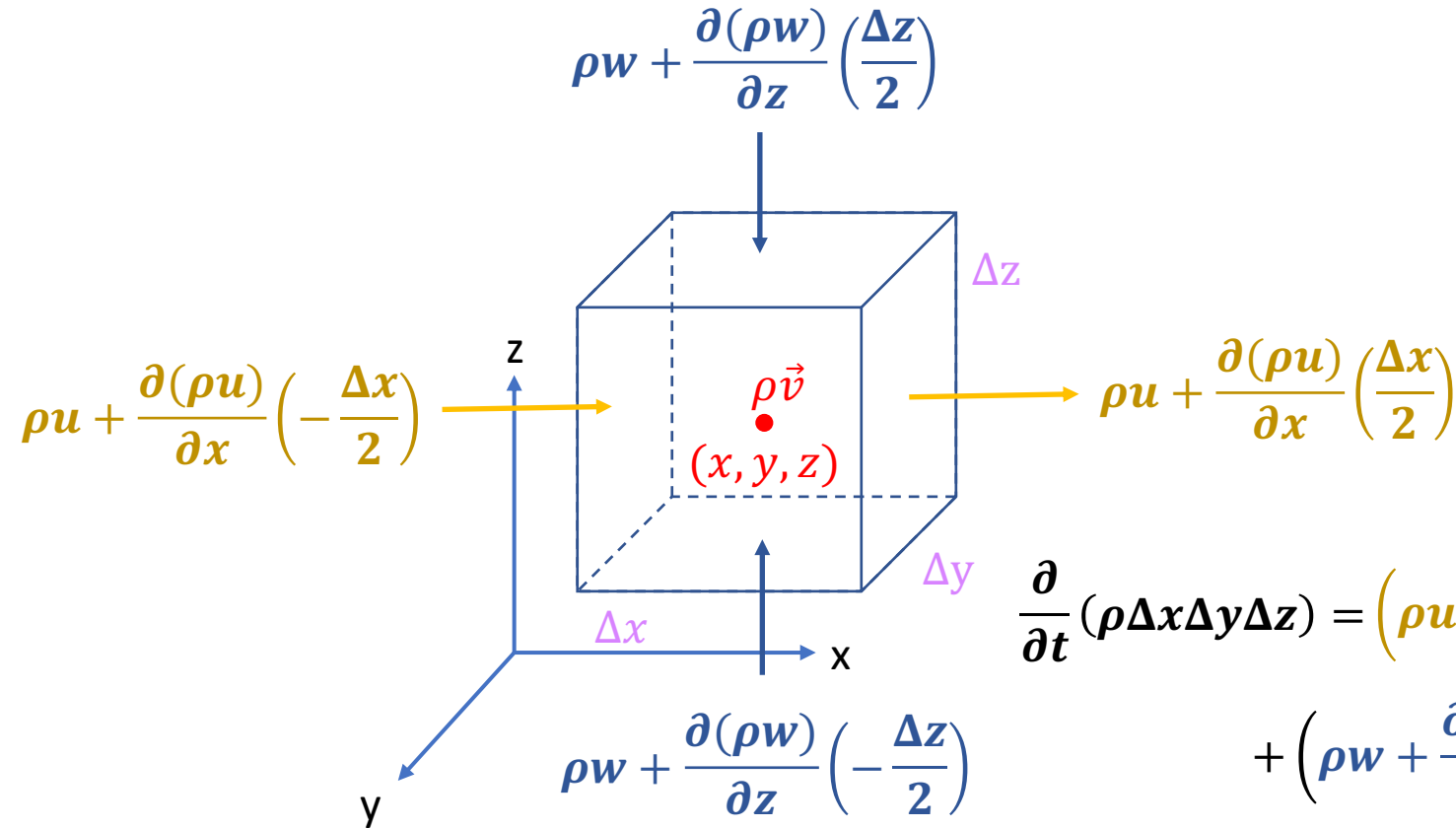
$$\rho u + \frac{\partial(\rho u)}{\partial x} \left(\frac{\Delta x}{2} \right)$$

Conservation of mass

Let's write a conservation law ...

Remember:

$$\left[\begin{array}{c} \text{Rate of inflow} \\ \text{to a system} \end{array} \right] - \left[\begin{array}{c} \text{Rate of outflow} \\ \text{from a system} \end{array} \right] = \left[\begin{array}{c} \text{Rate of} \\ \text{accumulation} \end{array} \right]$$



$$\begin{aligned} \frac{\partial}{\partial t} (\rho \Delta x \Delta y \Delta z) = & \left(\rho u + \frac{\partial(\rho u)}{\partial x} \left(-\frac{\Delta x}{2} \right) \Delta y \Delta z \right) - \left(\rho u + \frac{\partial(\rho u)}{\partial x} \left(\frac{\Delta x}{2} \right) \Delta y \Delta z \right) \\ & + \left(\rho w + \frac{\partial(\rho w)}{\partial z} \left(-\frac{\Delta z}{2} \right) \Delta x \Delta y \right) - \left(\rho w + \frac{\partial(\rho w)}{\partial z} \left(\frac{\Delta z}{2} \right) \Delta x \Delta y \right) \\ & + \left(\rho v + \frac{\partial(\rho v)}{\partial y} \left(-\frac{\Delta y}{2} \right) \Delta x \Delta z \right) - \left(\rho v + \frac{\partial(\rho v)}{\partial y} \left(\frac{\Delta y}{2} \right) \Delta x \Delta z \right) \end{aligned}$$

Conservation of mass

$$\begin{aligned}
 \frac{\partial}{\partial t}(\rho \Delta x \Delta y \Delta z) &= \left(\cancel{\rho u} + \frac{\partial(\rho u)}{\partial x} \left(-\frac{\Delta x}{2} \right) \Delta y \Delta z \right) - \left(\cancel{\rho u} + \frac{\partial(\rho u)}{\partial x} \left(\frac{\Delta x}{2} \right) \Delta y \Delta z \right) \\
 &+ \left(\cancel{\rho w} + \frac{\partial(\rho w)}{\partial z} \left(-\frac{\Delta z}{2} \right) \Delta x \Delta y \right) - \left(\cancel{\rho w} + \frac{\partial(\rho w)}{\partial z} \left(\frac{\Delta z}{2} \right) \Delta x \Delta y \right) \\
 &+ \left(\cancel{\rho v} + \frac{\partial(\rho v)}{\partial y} \left(-\frac{\Delta y}{2} \right) \Delta x \Delta z \right) - \left(\cancel{\rho v} + \frac{\partial(\rho v)}{\partial y} \left(\frac{\Delta y}{2} \right) \Delta x \Delta z \right) \\
 &= \left[-\frac{\partial(\rho u)}{\partial x} - \frac{\partial(\rho v)}{\partial y} - \frac{\partial(\rho w)}{\partial z} \right] \cancel{\Delta x \Delta y \Delta z}
 \end{aligned}$$

$$\frac{\partial \rho}{\partial t} = -\frac{\partial(\rho u)}{\partial x} - \frac{\partial(\rho v)}{\partial y} - \frac{\partial(\rho w)}{\partial z}$$

$$\frac{\partial \rho}{\partial t} + \frac{\partial(\rho u)}{\partial x} + \frac{\partial(\rho v)}{\partial y} + \frac{\partial(\rho w)}{\partial z} = 0$$

Continuity equation or
conservation of mass

Conservation of mass

$$\frac{\partial \rho}{\partial t} + \frac{\partial(\rho u)}{\partial x} + \frac{\partial(\rho v)}{\partial y} + \frac{\partial(\rho w)}{\partial z} = 0$$

Recall the gradient or “DEL” operator

$$\nabla(\cdot) = \frac{\partial(\cdot)}{\partial x} \hat{i} + \frac{\partial(\cdot)}{\partial y} \hat{j} + \frac{\partial(\cdot)}{\partial z} \hat{k}$$

So, if $\vec{v} = u\hat{i} + v\hat{j} + w\hat{k}$, then

$$\nabla \cdot \vec{v} = \frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} + \frac{\partial w}{\partial z}$$

Dot product

Scalar

$$\frac{\partial \rho}{\partial t} + \nabla \cdot (\rho \mathbf{v}) = 0$$

Incompressible flow

$\rho = \text{constant}$ (in some cases)

$$\cancel{\frac{\partial \rho}{\partial t}} + \nabla \cdot (\rho \mathbf{v}) = 0$$

$$\cancel{\rho}(\nabla \cdot \mathbf{v}) = 0$$

$$\nabla \cdot \mathbf{v} = 0$$

Steady-state

$$\frac{\partial \rho}{\partial t} = 0 \text{ (no changes with time)}$$

$$\nabla \cdot (\rho \mathbf{v}) = 0$$

Conservation of a scalar quantity

$$\underbrace{\left[\begin{array}{c} \text{Time rate of} \\ \text{change of} \\ \text{tracer} \end{array} \right]} = \underbrace{\left[\begin{array}{c} \text{Rate of} \\ \text{tracer inflow} \\ \text{to CV} \end{array} \right]} - \underbrace{\left[\begin{array}{c} \text{Rate of} \\ \text{tracer outflow} \\ \text{to CV} \end{array} \right]} + \underbrace{\left[\begin{array}{c} \text{source} \\ \text{term} \end{array} \right]} - \underbrace{\left[\begin{array}{c} \text{Turbulent} \\ \text{tracer flux} \end{array} \right]}$$

$$\underbrace{\frac{\partial c}{\partial t}}_{\text{Accumulation}} = \underbrace{- \left(u \frac{\partial c}{\partial x} + v \frac{\partial c}{\partial y} + w \frac{\partial c}{\partial z} \right)}_{\text{Advection}} + \underbrace{+ S_c}_{\text{Source term}} - \underbrace{\frac{\partial \overline{u'c'}}{\partial x} - \frac{\partial \overline{v'c'}}{\partial y} - \frac{\partial \overline{w'c'}}{\partial z}}_{\text{Turbulent mixing}}$$

Conservation of heat

$$\underbrace{\left(\begin{array}{c} \text{Time rate of} \\ \text{change of} \\ \text{heat} \end{array} \right)} = \underbrace{\left(\begin{array}{c} \text{Rate of} \\ \text{heat inflow} \\ \text{to CV} \end{array} \right)} - \underbrace{\left(\begin{array}{c} \text{Rate of} \\ \text{heat outflow} \\ \text{to CV} \end{array} \right)} + \underbrace{\left(\begin{array}{c} \text{source} \\ \text{term} \end{array} \right)} + \underbrace{\left(\begin{array}{c} \text{Turbulent} \\ \text{heat flux} \end{array} \right)}$$

$$\underbrace{\frac{\partial \theta}{\partial t}} = \underbrace{- \left(u \frac{\partial \theta}{\partial x} + v \frac{\partial \theta}{\partial y} + w \frac{\partial \theta}{\partial z} \right)} - \underbrace{\frac{1}{\rho c_p} \left(\frac{\partial Q_x^*}{\partial x} + \frac{\partial Q_y^*}{\partial y} + \frac{\partial Q_z^*}{\partial z} \right)} - \underbrace{\frac{L_v E}{\rho c_p}}$$

Accumulation Advection Radiation divergence Latent heat release

θ : potential temperature

L : latent heat

c_p : specific heat

E : phase change of moisture

Q^* : net radiation flux

$$\underbrace{- \frac{\partial \overline{u'\theta'}}{\partial x} - \frac{\partial \overline{v'\theta'}}{\partial y} - \frac{\partial \overline{w'\theta'}}{\partial z}}_{\text{Turbulent heat flux}}$$

Conservation of momentum

Newton's lex secunda (law of motion)

(Example: zonal momentum equation)

$$\underbrace{\left(\begin{array}{c} \text{Time rate of} \\ \text{change of} \\ \text{momentum} \end{array} \right)} = \underbrace{\left(\begin{array}{c} \text{Rate of} \\ \text{momentum} \\ \text{inflow to CV} \end{array} \right)} - \underbrace{\left(\begin{array}{c} \text{Rate of} \\ \text{momentum} \\ \text{outflow to CV} \end{array} \right)} + \underbrace{\left(\begin{array}{c} \text{Sum of forces} \\ \text{acting on CV} \end{array} \right)}$$

$$\underbrace{\frac{\partial u}{\partial t}}_{\text{Accumulation}} = \underbrace{- \left(u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} + w \frac{\partial u}{\partial z} \right)}_{\text{Advection}} \underbrace{- g_x}_{\text{Gravitational force}} \underbrace{- \frac{1}{\rho} \frac{\partial p}{\partial x}}_{\text{Pressure force}} \underbrace{- \frac{\partial \overline{u'u'}}{\partial x} - \frac{\partial \overline{u'v'}}{\partial y} - \frac{\partial \overline{u'w'}}{\partial z}}_{\text{Turbulent flux divergence}}$$

“Navier-Stokes Equation”

Conservation of momentum

$$\underbrace{\rho \frac{\partial u}{\partial t}}_{\text{Accumulation}} = \underbrace{-\rho \left(u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} + w \frac{\partial u}{\partial z} \right)}_{\text{Flow}} \underbrace{-\rho g_x}_{\text{Gravitational force}} \underbrace{-\frac{\partial p}{\partial x}}_{\text{Pressure force}} \underbrace{-\frac{\partial \overline{u'u'}}{\partial x} + \frac{\partial \overline{u'v'}}{\partial y} + \frac{\partial \overline{u'w'}}{\partial z}}_{\text{Turbulent flux divergence}}$$

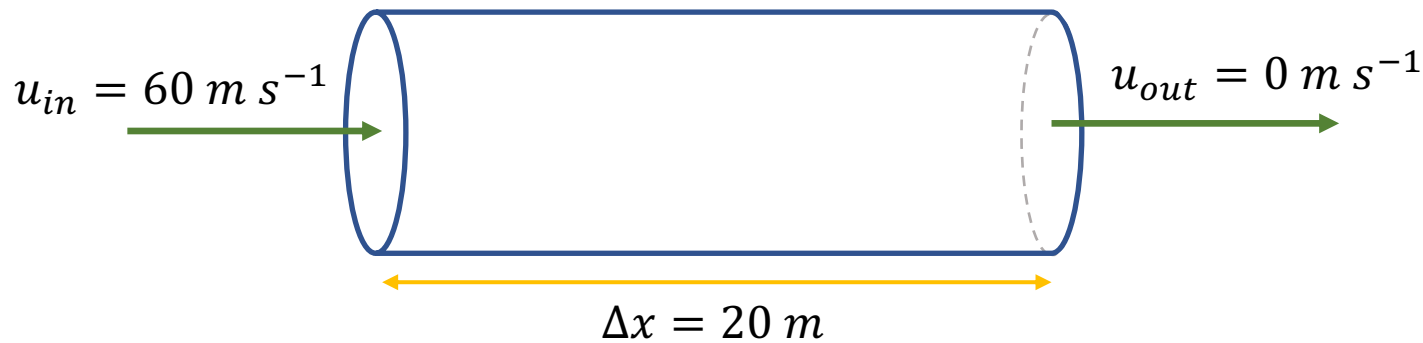
We can also write the Navier-Stokes equation in VECTOR FORM

$$\rho \frac{D\mathbf{u}}{Dt} = \rho \left(\frac{\partial \mathbf{u}}{\partial t} + \mathbf{u}(\nabla \cdot \mathbf{u}) \right) = -\nabla p + \rho \mathbf{g} + \nabla(\nabla \cdot \boldsymbol{\tau})$$

where \mathbf{u} is the velocity vector (not the u component)

Example: Conservation of air mass

Hurricane-force winds of 60 m s^{-1} blow into an west-facing entrance of a 20 m long pedestrian tunnel. The door at the other end of the tunnel is closed. The initial air density in the tunnel is 1.2 kg m^{-3} . Find the rate of air density increase in the tunnel.



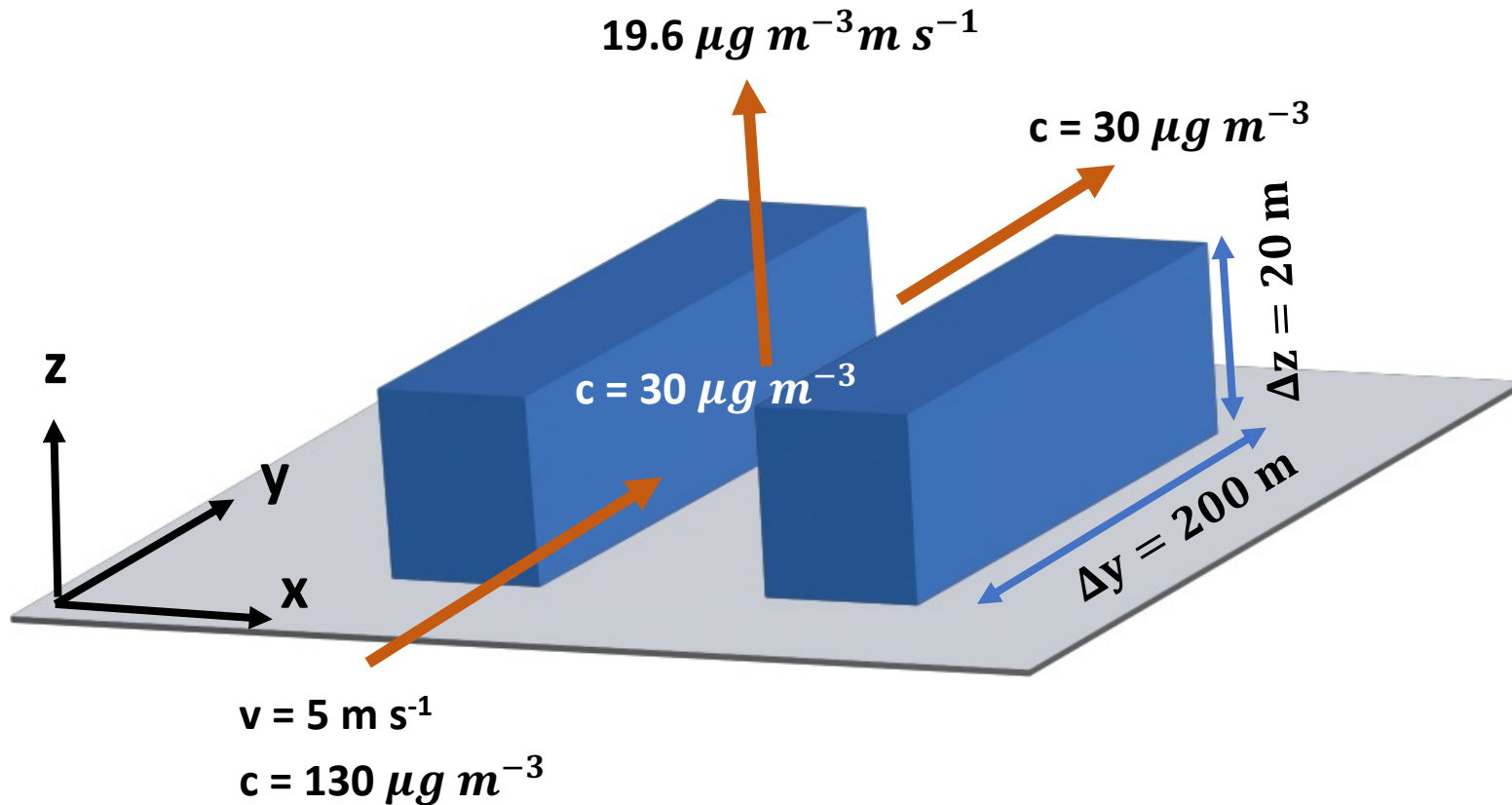
$$\frac{\partial \rho}{\partial t} = -\frac{\partial(\rho u)}{\partial x} - \cancel{\frac{\partial(\rho v)}{\partial y}} - \cancel{\frac{\partial(\rho w)}{\partial z}}$$

Solution:

$$\begin{aligned}\frac{\partial \rho}{\partial t} &= -\frac{\partial(\rho u)}{\partial x} \\ &= -(1.2 \text{ kg m}^{-3}) \frac{0 - 60 \text{ m s}^{-1}}{20 \text{ m}} \\ &= +3.6 \text{ kg m}^{-3} \text{ s}^{-1}\end{aligned}$$

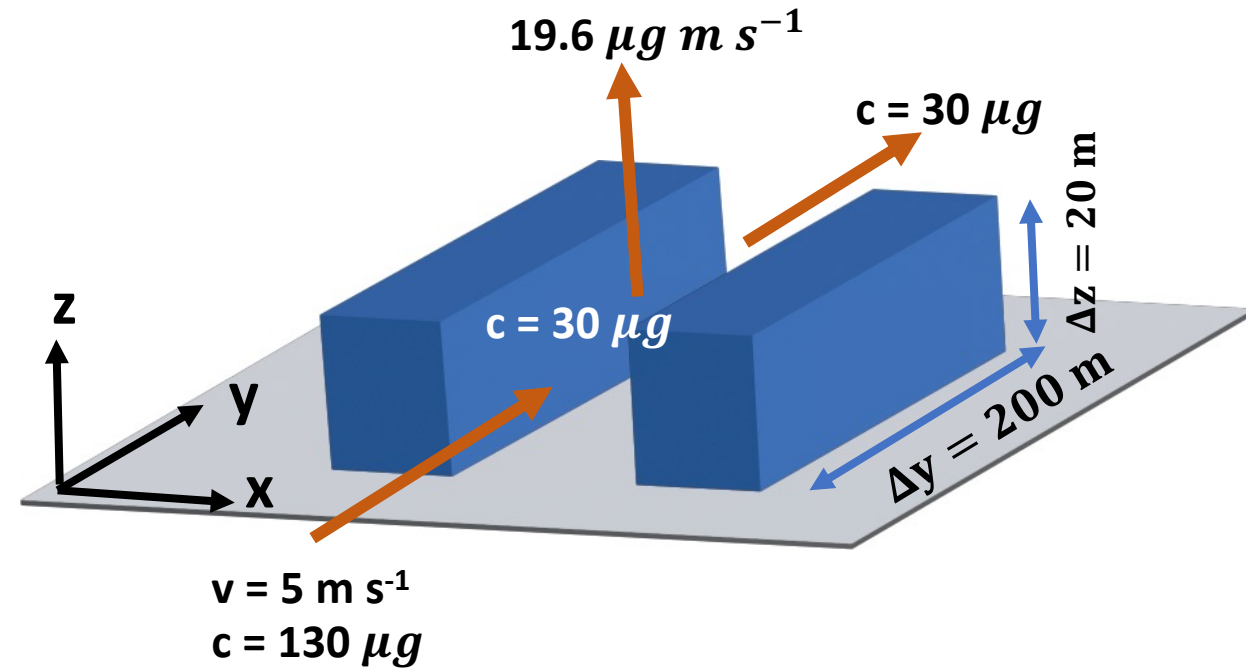
Example: Conservation of particle matter

In a narrow street canyon, a fine particle matter load of $30 \mu\text{g m}^{-3}$ is measured. A moderate wind blows from the south, advecting further particles from a busy road. The concentration at the road is $130 \mu\text{g m}^{-3}$. Turbulent mixing causes $19.6 \mu\text{g} \cdot \text{m} \cdot \text{s}^{-1}$ to be entrained upwards. Does the concentration exceed the critical threshold value of $150 \mu\text{g m}^{-3}$ after one hour?



$$\frac{\partial c}{\partial t} = \underbrace{- \left(\cancel{u \frac{\partial c}{\partial x}} + v \frac{\partial c}{\partial y} + \cancel{w \frac{\partial c}{\partial z}} \right)}_{\text{Advection}} + \underbrace{S_c}_{\text{Source term}} - \underbrace{\left(\cancel{\frac{\partial \overline{u'c'}}{\partial x}} - \cancel{\frac{\partial \overline{v'c'}}{\partial y}} - \frac{\partial \overline{w'c'}}{\partial z} \right)}_{\text{Turbulent mixing}}$$

Example: Conservation of particle matter



Solution:

a) Calculate the change of particle matter

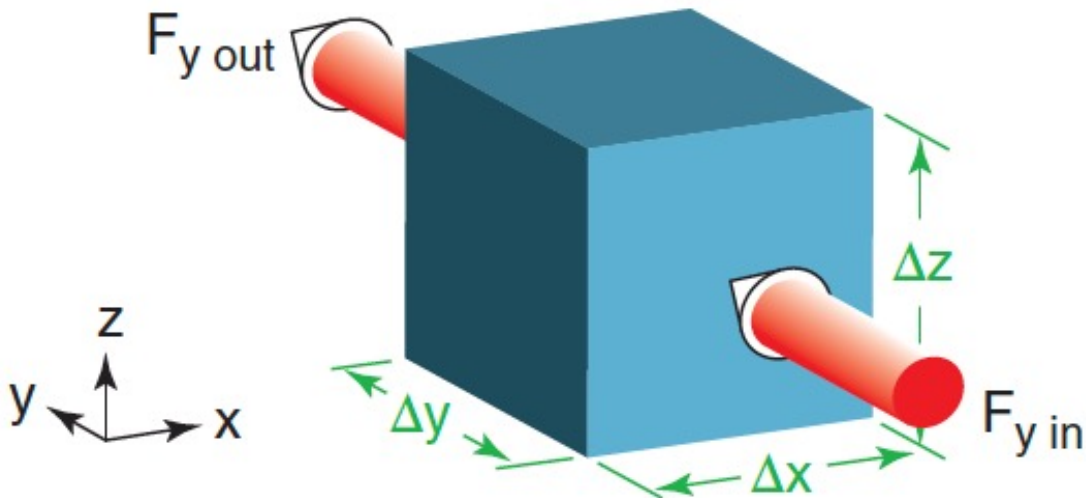
$$\begin{aligned} \frac{\partial c}{\partial t} &= -v \frac{\partial c}{\partial y} - \frac{\partial \overline{w' c'}}{\partial z} \\ &= -2 \text{ (m s}^{-1}\text{)} \frac{(30 - 130) \mu\text{g}}{200 \text{ m}} \\ &\quad - \frac{19.6 \text{ } \mu\text{g m s}^{-1}}{20 \text{ m}} \\ &= \mathbf{0.02 \text{ } \mu\text{g s}^{-1}} \end{aligned}$$

b) Calculate the change of particle matter in one hour.

$$\frac{\partial c}{\partial t} = \frac{c_t - c_0}{dt} = 0.02 \text{ } \mu\text{g s}^{-1} \quad c_t = c_0 + (0.02 \text{ } \mu\text{g s}^{-1}) \cdot dt = \mathbf{102 \text{ } \mu\text{g}}$$

Example: Heat advection

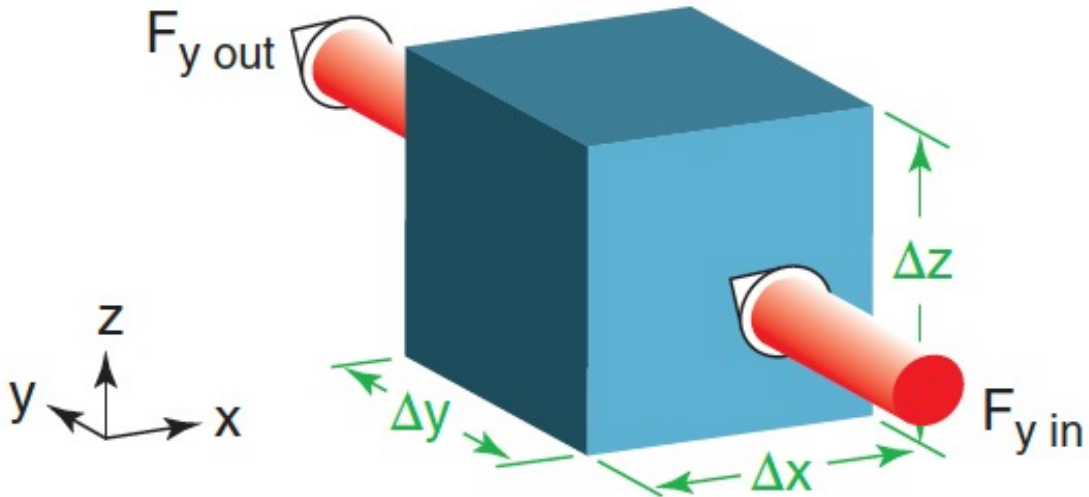
The cube of air below has $\theta = 12^\circ\text{C}$ along its south side, but smoothly increases in temperature to 15°C on the north side. This 100 km square cube is advecting toward the north at 25 km/hour. What warming rate at a fixed thermometer can be attributed to temperature advection?



$$\frac{\partial \theta}{\partial t} = \underbrace{- \left(u \frac{\partial \theta}{\partial x} + v \frac{\partial \theta}{\partial y} + w \frac{\partial \theta}{\partial z} \right)}_{\text{Advection}} - \underbrace{\frac{1}{\rho c_p} \left(\frac{\partial Q_x^*}{\partial x} + \frac{\partial Q_y^*}{\partial y} + \frac{\partial Q_z^*}{\partial z} \right)}_{\text{Radiation divergence}} - \underbrace{\frac{L_v E}{\rho c_p}}_{\text{Latent heat release}} - \underbrace{\left(\frac{\partial \overline{u'\theta'}}{\partial x} + \frac{\partial \overline{v'\theta'}}{\partial y} + \frac{\partial \overline{w'\theta'}}{\partial z} \right)}_{\text{Turbulent heat flux}}$$

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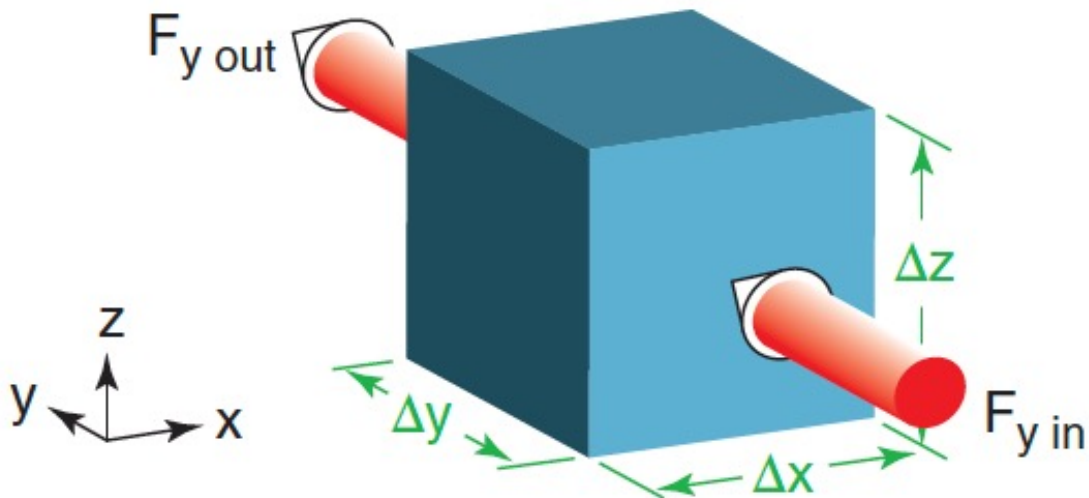


$$\begin{aligned}\frac{\partial \theta}{\partial t} &= -v \frac{\partial \theta}{\partial y} \\ &= -(25 \text{ km h}^{-1}) \cdot \frac{15^{\circ}\text{C} - 12^{\circ}\text{C}}{100 \text{ km}} \\ &= -0.75^{\circ}\text{C h}^{-1}\end{aligned}$$

Example: Turbulent heat flux

In the figure below, suppose that the incoming heat flux from the south is 5 W m^{-2} , and the outgoing on the north face of the cube is 7 W m^{-2} .

- Convert these fluxes to kinematic units.
- What is the value of the kinematic flux gradient?
- Calculate the warming rate of air in the cube, assuming the cube has zero humidity and is at a fixed altitude where air density is 1 kg m^{-3} . The cube of air is 10 m on each side.



$$\frac{\partial \theta}{\partial t} = \underbrace{- \left(u \frac{\partial \theta}{\partial x} + v \frac{\partial \theta}{\partial y} + w \frac{\partial \theta}{\partial z} \right)}_{\text{Advection}} - \underbrace{\frac{1}{\rho c_p} \left(\frac{\partial Q_x^*}{\partial x} + \frac{\partial Q_y^*}{\partial y} + \frac{\partial Q_z^*}{\partial z} \right)}_{\text{Radiation divergence}} - \underbrace{\frac{L_v E}{\rho c_p}}_{\text{Latent heat release}} - \underbrace{\frac{\partial \overline{u'\theta'}}{\partial x} + \frac{\partial \overline{v'\theta'}}{\partial y} + \frac{\partial \overline{w'\theta'}}{\partial z}}_{\text{Turbulent heat flux}}$$

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In the figure below, suppose that the incoming heat flux from the south is 5 W m^{-2} , and the outgoing on the north face of the cube is 7 W m^{-2} .

- (a) Convert these fluxes to kinematic units.
- (b) What is the value of the kinematic flux gradient?
- (c) Calculate the warming rate of air in the cube, assuming the cube has zero humidity and is at a fixed altitude where air density is 1 kg m^{-3} . The cube of air is 10 m on each side.

$$\begin{array}{c} \text{[K]} \swarrow \\ \frac{\partial \theta}{\partial t} = - \frac{\partial \overline{v' \theta'}}{\partial y} \nwarrow \\ \swarrow \text{[s]} \quad \quad \quad \searrow \text{[m]} \end{array}$$

[K m s⁻¹]

- (a) **Convert heat flux [W m^{-2}] to the kinetic heat flux [K m s^{-1}]**

$$\begin{aligned} \overline{v' \theta'} &= \frac{5 \text{ W m}^{-2}}{\rho \cdot c_p} \\ &= \frac{5 \text{ W m}^{-2}}{1 \text{ kg m}^{-3} \cdot 1004 \text{ J kg}^{-1} \text{ K}^{-1}} \\ &= \underline{4.98 \cdot 10^{-3} \text{ K} \cdot \text{m} \cdot \text{s}^{-1}} \end{aligned}$$

$$\begin{aligned} \overline{v' \theta'} &= \frac{7 \text{ W m}^{-2}}{1 \text{ kg m}^{-3} \cdot 1004 \text{ J kg}^{-1} \text{ K}^{-1}} \\ &= \underline{6.97 \cdot 10^{-3} \text{ K} \cdot \text{m} \cdot \text{s}^{-1}} \end{aligned}$$

Example: Turbulent heat flux

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- (a) Convert these fluxes to kinematic units.
- (b) What is the value of the kinematic flux gradient?
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$$\begin{array}{c} \text{[K]} \swarrow \\ \frac{\partial \theta}{\partial t} = - \frac{\partial \overline{v' \theta'}}{\partial y} \nwarrow \\ \swarrow \text{[s]} \quad \quad \quad \text{[m]} \end{array}$$

[K m s⁻¹]

(b) What is the kinematic flux gradient?

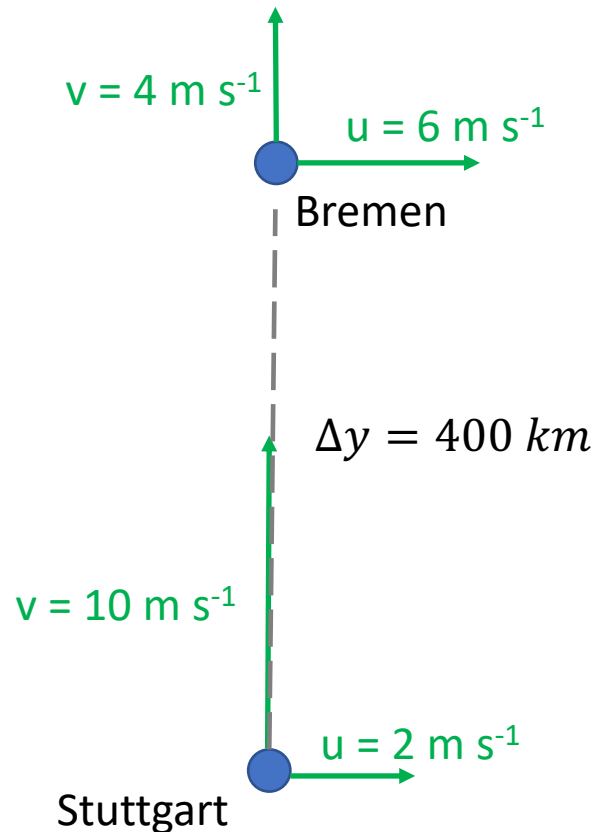
$$\begin{aligned} \frac{\partial \overline{v' \theta'}}{\partial y} &= \frac{[6.97 \cdot 10^{-3} - 4.98 \cdot 10^{-3}]}{[10 - 0]} \\ &= -1.99 \cdot 10^{-4} \text{ K} \cdot \text{s}^{-1} \end{aligned}$$

(c) Calculate the warming rate.

$$\frac{\partial \theta}{\partial t} = \frac{\partial \overline{v' \theta'}}{\partial y} = -1.99 \cdot 10^{-4} \text{ K} \cdot \text{s}^{-1}$$

Example: Advection of momentum

Bremen is about 400 km north of Stuttgart. In Bremen the wind components (U, V) are (6, 4) m s⁻¹, while in Stuttgart they are (2, 10) m s⁻¹. What is the value of the advective force per mass?



$$\frac{\partial u}{\partial t} = \underbrace{- \left(\cancel{u \frac{\partial u}{\partial x}} + v \frac{\partial u}{\partial y} + \cancel{w \frac{\partial u}{\partial z}} \right)}_{\text{Advection}} + \underbrace{\cancel{g_x}}_{\text{Gravitational force}} - \underbrace{\cancel{\frac{1}{\rho} \frac{\partial p}{\partial x}}}_{\text{Pressure force}} - \underbrace{\cancel{\frac{\partial \overline{u'u'}}{\partial x}} - \frac{\partial \overline{u'v'}}{\partial y} - \frac{\partial \overline{u'w'}}{\partial z}}_{\text{Turbulent flux divergence}}$$

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$$\frac{\partial u}{\partial t} = -v \frac{\partial u}{\partial y} \quad \frac{\partial v}{\partial t} = -v \frac{\partial v}{\partial y}$$

Gradients:

$$\frac{\partial u}{\partial y} = \frac{(6 - 2 \text{ m s}^{-1})}{400,000 \text{ m}} = 1.0 \cdot 10^{-5} \text{ s}^{-1}$$

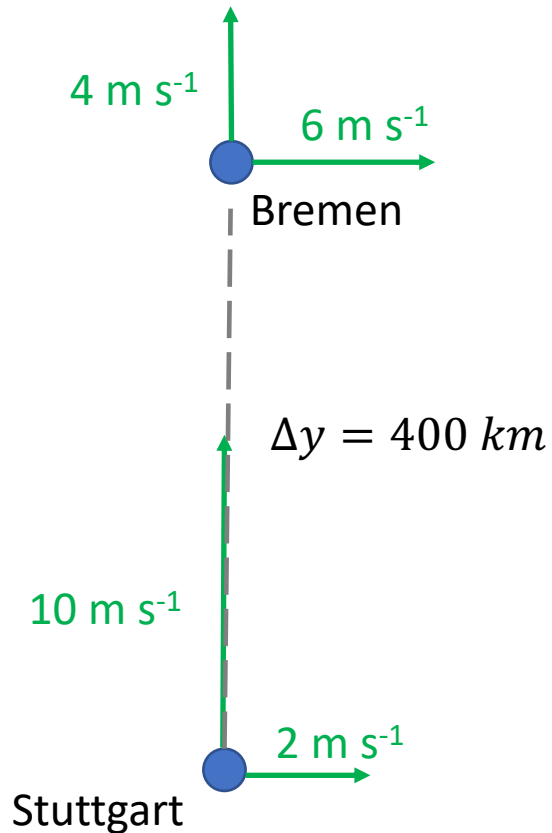
$$\frac{\partial v}{\partial y} = \frac{(4 - 10 \text{ m s}^{-1})}{400,000 \text{ m}} = -1.5 \cdot 10^{-5} \text{ s}^{-1}$$

Average wind speed:

$$u = \frac{(6 + 2 \text{ m s}^{-1})}{2} = 4 \text{ m s}^{-1} \quad v = \frac{(4 + 10 \text{ m s}^{-1})}{2} = 7 \text{ m s}^{-1}$$

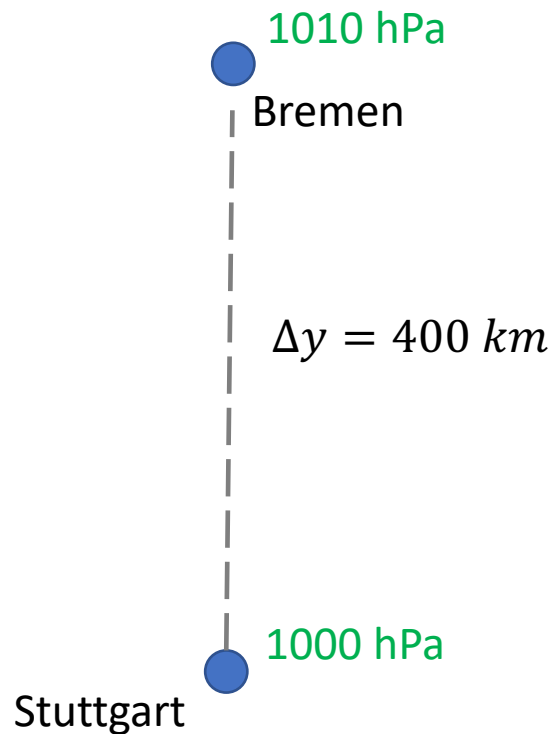
$$\begin{aligned} \frac{\partial u}{\partial t} &= -(7 \text{ m s}^{-1}) \cdot (1.0 \cdot 10^{-5} \text{ s}^{-1}) \\ &= -7 \cdot 10^{-5} \text{ m} \cdot \text{s}^{-2} \end{aligned}$$

$$\begin{aligned} \frac{\partial v}{\partial t} &= -(7 \text{ m s}^{-1}) \cdot (-1.5 \cdot 10^{-5} \text{ s}^{-1}) \\ &= 1.05 \cdot 10^{-4} \text{ m} \cdot \text{s}^{-2} \end{aligned}$$



Example: Advection of momentum

Bremen is about 400 km north of Stuttgart. In Bremen the pressure is 1010 hPa, while in Stuttgart the pressure is 1000 hPa. Find the pressure gradient force? (let $\rho = 1.1 \text{ kg m}^{-3}$)



$$\frac{\partial v}{\partial t} = \underbrace{- \left(u \frac{\partial v}{\partial x} + v \frac{\partial v}{\partial y} + w \frac{\partial v}{\partial z} \right)}_{\text{Advection}}$$

$$\underbrace{-g_y}_{\text{Gravitational force}}$$

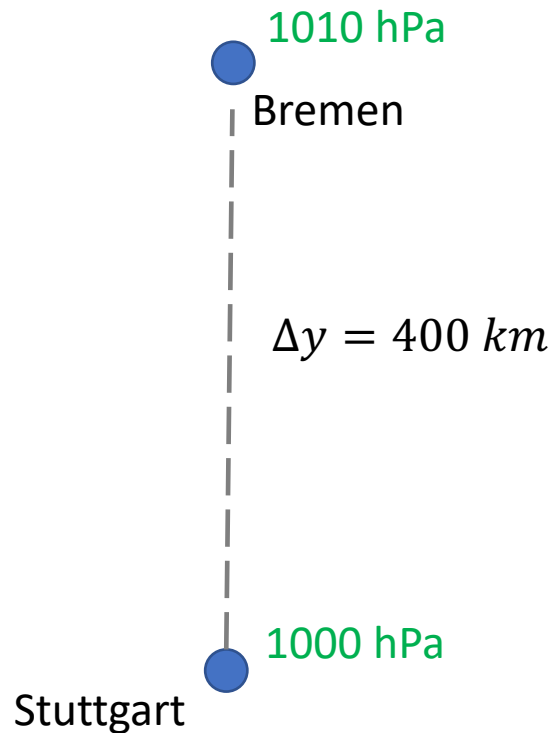
$$\underbrace{-\frac{1}{\rho} \frac{\partial p}{\partial y}}_{\text{Pressure force}}$$

$$\underbrace{- \left(\frac{\partial \overline{v'u'}}{\partial x} + \frac{\partial \overline{v'v'}}{\partial y} + \frac{\partial \overline{v'w'}}{\partial z} \right)}_{\text{Turbulent flux divergence}}$$

Example: Advection of momentum

Bremen is about 400 km north of Stuttgart. In Bremen the pressure is 1010 hPa, while in Stuttgart the pressure is 1000 hPa. Find the pressure gradient force? (let $\rho = 1.1 \text{ kg m}^{-3}$)

Solution:



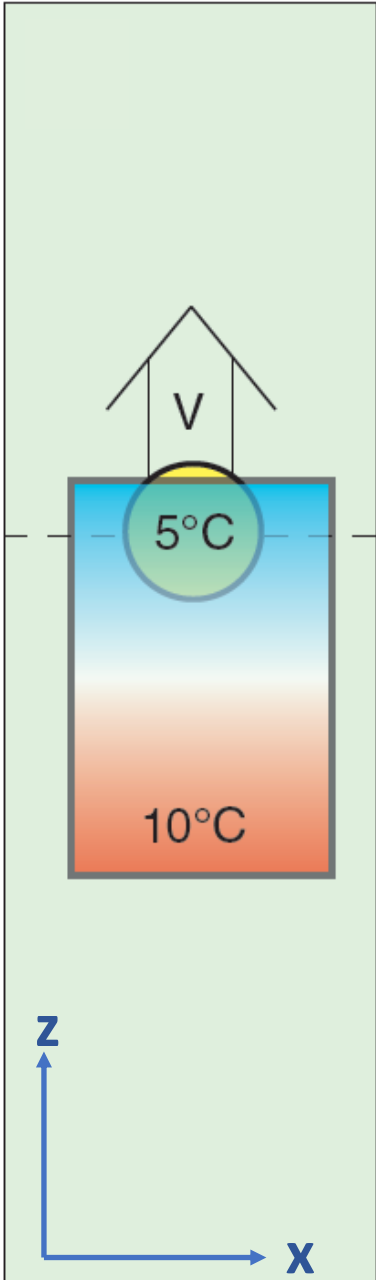
$$\frac{\partial v}{\partial t} = -\frac{1}{\rho} \frac{\partial p}{\partial y}$$

$$= -\frac{1}{1.1 \text{ kg m}^{-3}} \frac{(101,000 - 100,000) \text{ Pa}}{(400,000 - 0) \text{ m}}$$

$$= -2.27 \cdot 10^{-3} \text{ m} \cdot \text{s}^{-2}$$

(hint: $1 \text{ Pa} = 1 \text{ kg m}^{-1} \text{ s}^{-2}$)

Homework: Heat advection



Homework 1: Given the left figure, assume that higher in the figure corresponds to higher in the atmosphere. Suppose that the 5°C air is at a relative altitude that is 500 m higher than that of the 10°C air. If the updraft is 500 m/(10 hours), what is the temperature at the thermometer (yellow circle) after 10 hours?

Homework 2: The potential temperature of the air increases 5°C per 100 km distance east. If an east wind of 20 m s⁻¹ is blowing, find the advective flux gradient, and the temperature change associated with this advection.

Homework 3: Suppose that the turbulent heat flux decreases linearly with height according to $\overline{w'\theta'} = a - b \cdot z$, where $a = 0.3$ (K ms⁻¹) and $b = 3 \cdot 10^{-4}$ (K s⁻¹). If the initial temperature profile is an arbitrary shape, then what will be the shape of the final profile on hour later? Neglecting subsidence, radiation, latent heating, and assume horizontal heterogeneity.

Homework 4: If a horizontal wind of 10 m/s is advecting drier air into a region, where the horizontal moisture gradient is (5 g_{water}/kg_{air})/100 km, then what vertical gradient of turbulent moisture flux in the boundary layer is required to maintain a steady-state specific humidity? Assume there is no body source of moisture.

Homework: Heat conservation

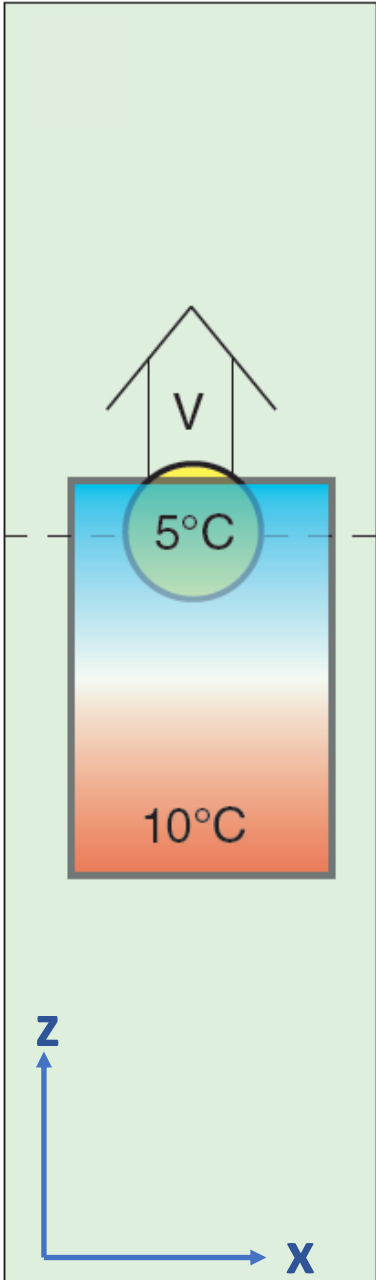
Homework 5: Find the rate of temperature change ($^{\circ}\text{C h}^{-1}$) with no internal heat source, given the kinematic flux divergence values below. Assume $\Delta x = \Delta y = \Delta z = 1 \text{ km}$.

	$\overline{u'\theta'} (K \cdot m \cdot s^{-1})$	$\overline{v'\theta'} (K \cdot m \cdot s^{-1})$	$\overline{w'\theta'} (K \cdot m \cdot s^{-1})$
a)	1	2	3
b)	1	2	-3
c)	1	-2	3
d)	1	-2	-3

Homework 6: Given the wind and temperature gradient, find the value of the kinematic advective flux gradient ($^{\circ}\text{C h}^{-1}$).

	$w (m \cdot s^{-1})$	$\partial\theta/\partial z (^{\circ}\text{C} \cdot \text{km}^{-1})$
a)	5	-2
b)	5	2
c)	10	-5
d)	10	-10

Homework: Heat advection



Homework 1: Given the left figure, assume that higher in the figure corresponds to higher in the atmosphere. Suppose that the 5°C air is at a relative altitude that is 500 m higher than that of the 10°C air. If the updraft is 500 m/(10 hours), what is the temperature at the thermometer (yellow circle) after 10 hours?

$$\frac{\partial \theta}{\partial t} = - \left(u \frac{\partial \theta}{\partial x} + v \frac{\partial \theta}{\partial y} + w \frac{\partial \theta}{\partial z} \right) - \frac{1}{\rho c_p} \left(\frac{\partial Q_x^*}{\partial x} + \frac{\partial Q_y^*}{\partial y} + \frac{\partial Q_z^*}{\partial z} \right) - \frac{L_v E}{\rho c_p} - \frac{\partial \overline{u' \theta'}}{\partial x} - \frac{\partial \overline{v' \theta'}}{\partial y} - \frac{\partial \overline{w' \theta'}}{\partial z}$$

$$\frac{\partial \theta}{\partial t} = \frac{\theta_{t+1} - \theta_t}{dt} \quad \longrightarrow \quad \theta_{n+1} = \theta_t - w \frac{\partial \theta}{\partial z} dt$$

$$= 5 \text{ [}^\circ\text{C]} - 500 \text{ [m } 10\text{h}^{-1}] \frac{5 - 10 \text{ [}^\circ\text{C}]}{500 \text{ [m]}} \cdot 10 \text{ [h]} = \mathbf{10 \text{ [}^\circ\text{C]}}$$

BUT: The air that is initially 10°C will adiabatically cool 9.8°C/km of rise. Here, it rises only 0.5 km in the 10 h, so it cools 9.8°C/2 = 4.9°C.

Its final temperature is 10 °C - 4.9°C = 5.1 °C $\longrightarrow \frac{\partial \theta}{\partial t} = -w \left(\frac{\partial \theta}{\partial z} + \Gamma_d \right)$

Homework: Heat advection

Homework 2: The potential temperature of the air increases 5°C per 100 km distance east. If an east wind of 20 m s⁻¹ is blowing, find the advective flux gradient, and the temperature change associated with this advection.

$$\frac{\partial \theta}{\partial t} = - \left(u \frac{\partial \theta}{\partial x} + v \frac{\partial \theta}{\partial y} + w \frac{\partial \theta}{\partial z} \right) - \frac{1}{\rho c_p} \left(\frac{\partial Q_x^*}{\partial x} + \frac{\partial Q_y^*}{\partial y} + \frac{\partial Q_z^*}{\partial z} \right) - \frac{L_v E}{\rho c_p} - \frac{\partial \overline{u' \theta'}}{\partial x} - \frac{\partial \overline{v' \theta'}}{\partial y} - \frac{\partial \overline{w' \theta'}}{\partial z}$$

$$\frac{\partial \theta}{\partial t} = -v \frac{\partial \theta}{\partial y}$$

$$= -(-20 \text{ [m s}^{-1}\text{]}) \frac{5 \text{ [}^\circ\text{C]}}{100000 \text{ [m]}} = -20 \text{ [m s}^{-1}\text{]} \cdot \mathbf{0.00005 \text{ [}^\circ\text{C m}^{-1}\text{]}}$$

$$= \mathbf{+0.001 \text{ [}^\circ\text{C s}^{-1}\text{]}}$$

Homework: Heat advection

Homework 3: Suppose that the turbulent heat flux decreases linearly with height according to $\overline{w'\theta'} = a - b \cdot z$, where $a = 0.3 \text{ (K ms}^{-1}\text{)}$ and $b = 3 \cdot 10^{-4} \text{ (K s}^{-1}\text{)}$. If the initial temperature profile is an arbitrary shape, then what will be the shape of the final profile on hour later? Neglecting subsidence, radiation, latent heating, and assume horizontal heterogeneity.

$$\frac{\partial \theta}{\partial t} = - \left(u \frac{\partial \theta}{\partial x} + v \frac{\partial \theta}{\partial y} + w \frac{\partial \theta}{\partial z} \right) - \frac{1}{\rho c_p} \left(\frac{\partial Q_x^*}{\partial x} + \frac{\partial Q_y^*}{\partial y} + \frac{\partial Q_z^*}{\partial z} \right) - \frac{L_v E}{\rho c_p} - \frac{\partial \overline{u'\theta'}}{\partial x} - \frac{\partial \overline{v'\theta'}}{\partial y} - \frac{\partial \overline{w'\theta'}}{\partial z}$$

$$\frac{\partial \theta}{\partial t} = - \frac{\partial \overline{w'\theta'}}{\partial z}$$

Pluggin in the expression for $\overline{w'\theta'}$ gives:

$$\frac{\partial \theta}{\partial t} = - \frac{\partial \overline{w'\theta'}}{\partial z} = - \frac{\partial (a - b \cdot z)}{\partial z} = +b$$

Answer: The answer is not a function of z ; hence, air at each height in the sounding warms at the same rate.

Integrating over time from $t = t_0$ to t gives: $\theta_t = \theta_0 + b(t - t_0)$

$$b(t - t_0) = 3 \cdot 10^{-4} \text{ [K s}^{-1}\text{]} \cdot 3600 \text{ [s]} = \mathbf{1.08 \text{ K}}$$

Homework: Heat advection

Homework 4: If a horizontal wind of 10 m/s is advecting drier air into a region, where the horizontal moisture gradient is $(5 \text{ g}_{\text{water}}/\text{kg}_{\text{air}})/100 \text{ km}$, then what vertical gradient of turbulent moisture flux in the boundary layer is required to maintain a steady-state specific humidity? Assume there is no body source of moisture.

$$\frac{\partial q}{\partial t} = - \left(u \frac{\partial q}{\partial x} + v \frac{\partial q}{\partial y} + w \frac{\partial q}{\partial z} \right) + S_q - \frac{\partial \overline{u'q'}}{\partial x} - \frac{\partial \overline{v'q'}}{\partial y} - \frac{\partial \overline{w'q'}}{\partial z}$$

Answer: Steady-state is defined as one where there are no local changes of a variable with time (i.e. $\frac{\partial(\cdot)}{\partial t} = 0$). Choose the x-axis to be aligned with the mean wind direction for simplicity gives

$$u \frac{\partial q}{\partial x} = - \frac{\partial \overline{w'q'}}{\partial z}$$

$$10 \text{ [m s}^{-1}\text{]} \cdot 5 \cdot 10^{-5} \text{ [g}_{\text{water}} \text{ kg}_{\text{air}}^{-1} \text{ m}^{-1}\text{]} = - \frac{\partial \overline{w'q'}}{\partial z}$$

$$- \frac{\partial \overline{w'q'}}{\partial z} = -5 \cdot 10^{-4} \left[\frac{\text{g}_{\text{water}}}{\text{kg}_{\text{air}} \cdot \text{s}} \right]$$