Solid Mechanics - Zak Olech - 9/24/2019

Variables (Alphbetical By Variable)

a	Constant; distance
A, B, C,	Forces; reactions
A, B, C, \ldots	Points
A , α	Area
b	Distance; width
c	Constant; distance; radius
C	Centroid
C_1 , C_2 ,	Constants of integration
C_P	Column stability factor
d	Distance; diameter; depth
D	Diameter
e	Distance; eccentricity; dilatation
E	Modulus of elasticity
f	Frequency; function
F	Force
F.S.	Factor of safety
G	Modulus of rigidity; shear modulus
h	Distance; height
Н	Force
H, J, K	Points
I, I_x, \ldots	Moment of inertia
I_{xy}	Product of inertia

J	Polar moment of inertia
k	Spring constant; shape factor; bulk
	modulus; constant
K	Stress concentration factor; torsional
	spring constant
l	Length; span
L	Length; span
L_e	Effective length
m	Mass
\mathbf{M}	Couple
M , M_x ,	Bending moment
M_D	Bending moment, dead load (LRFD)
M_L	Bending moment, live load (LRFD)
M_U	Bending moment, ultimate load (LRFD)
n	Number; ratio of moduli of elasticity;

normal direction

Force; concentrated load

Dead load (LRFD)

Live load (LRFD)

Pressure

P

P_U	Ultimate load (LRFD)
q	Shearing force per unit length; shear
	flow
Q	Force
Q	First moment of area
r	Radius; radius of gyration
\mathbf{R}	Force; reaction
R	Radius; modulus of rupture
S	Length
\boldsymbol{S}	Elastic section modulus
t	Thickness; distance; tangential
	deviation
\mathbf{T}	Torque
T	Temperature
u, v	Rectangular coordinates
u	Strain-energy density
U	Strain energy; work
\mathbf{v}	Velocity
\mathbf{V}	Shearing force
V	Volume; shear
w	Width; distance; load per unit length
W , W	Weight, load

\mathbf{V}	Shearing force
V	Volume; shear
w	Width; distance; load per unit length
\mathbf{W} , W	Weight, load
<i>x</i> , <i>y</i> , <i>z</i>	Rectangular coordinates; distance;
	displacements; deflections
\overline{x} , \overline{y} , \overline{z}	Coordinates of centroid
Z	Plastic section modulus
α , β , γ	Angles
α	Coefficient of thermal expansion;
	influence coefficient
γ	Shearing strain; specific weight
γ_D	Load factor, dead load (LRFD)
γ_L	Load factor, live load (LRFD)
δ	Deformation; displacement
ϵ	Normal strain
heta	Angle; slope
λ	Direction cosine
u	Poisson's ratio
ho	Radius of curvature; distance; density
σ	Normal stress
au	Shearing stress
ϕ	Angle; angle of twist; resistance factor
ω	Angular velocity

Conversion Factors

$$1hp = 550ft*lb/s = 6600 in*lb/s$$

(1)

General

SI Prefixes

Multiplication Factor	Prefix [†]	Symbol
$1\ 000\ 000\ 000\ 000 = 10^{12}$	tera	Т
$1\ 000\ 000\ 000 = 10^9$	giga	G
$1\ 000\ 000 = 10^6$	mega	M
$1\ 000 = 10^3$	kilo	k
$100 = 10^2$	hecto‡	h
$10 = 10^{1}$	deka‡	da
$0.1 = 10^{-1}$	deci‡	d
$0.01 = 10^{-2}$	centi‡	c
$0.001 = 10^{-3}$	milli	m
$0.000\ 001 = 10^{-6}$	micro	μ
$0.000\ 000\ 001 = 10^{-9}$	nano	n
$0.000\ 000\ 000\ 001 = 10^{-12}$	pico	р
$0.000\ 000\ 000\ 000\ 001 = 10^{-15}$	femto	p f
$0.000\ 000\ 000\ 000\ 000\ 001 = 10^{-18}$	atto	a

Chapter 1 - Concept of stress

Axial Loading: Normal Stress

$$\sigma = \frac{P}{A} \tag{2}$$

Transverse Forces and Shearing Stress

$$\tau_{\text{ave}} = \frac{P}{A} \tag{3}$$

Single and Double Shear

Single Shear

$$\tau_{\text{avg}} = \frac{P}{A} = \frac{F}{A} \tag{4}$$

Double Shear

$$\tau_{\text{avg}} = \frac{P}{A} = \frac{F/2}{A} = \frac{F}{2A} \tag{5}$$

Bearing Stress

$$\sigma_b = \frac{P}{A} = \frac{P}{td} \tag{6}$$

Method of Solution

- 1. Clear and precise statement of problem
- 2. Draw one or several free-body diagrams; used to write equilibrium equations
- 3. Think SMART. Strategy, Modeling, Analysis, and Reflect

Stresses on an Oblique Section

$$\sigma = \frac{P}{A_0} \cos^2 \theta \tag{7}$$

$$\tau = \frac{P}{A_0} sin\theta cos\theta \tag{8}$$

Stress Under General Loading **Factor of Safety**

Factor of safety = F.S. =
$$\frac{\text{ultimate load}}{\text{allowable load}}$$
 (9)

Chapter 2 - Stress and Strain -Axial Loading

Normal Strain

$$\epsilon = \frac{\delta}{L} \tag{10}$$

Hooke's Law and Modulus of Elasticity

$$\sigma = E\epsilon \tag{11}$$

Elastic Deformation Under Axial Loading

$$\delta = \frac{PL}{AE} \tag{12}$$

$$\delta = \Sigma = \frac{P_i L_i}{A_i E_i} \tag{13}$$

Problems with Temperature Change

$$\delta_T = \alpha(\Delta T)L \tag{14}$$

$$\epsilon_T = \alpha \Delta T \tag{15}$$

Lateral Strain and Poisson's Ratio

$$v = -\frac{\text{lateral strain}}{\text{axial strain}} \tag{16}$$

Multiaxial Loading

$$\epsilon_x = \frac{\sigma_x}{E} \tag{17}$$

$$\sigma_y = \sigma_x = -\frac{v\sigma_x}{E} \tag{18}$$

Generalized Hooke's law for multiaxial loading

$$\sigma_x = +\frac{\sigma_x}{E} - \frac{v\sigma_y}{E} - \frac{v\sigma_z}{E} \tag{19}$$

$$\sigma_y = -\frac{\sigma_x}{E} - \frac{v\sigma_y}{E} - \frac{v\sigma_z}{E}$$

$$\sigma_z = -\frac{\sigma_x}{E} - \frac{v\sigma_y}{E} + \frac{v\sigma_z}{E}$$
(20)

$$\sigma_z = -\frac{\sigma_x}{E} - \frac{v\sigma_y}{E} + \frac{v\sigma_z}{E} \tag{21}$$

Dilation

$$e = \frac{1 - 2v}{E} (\sigma_x + \sigma_y + \sigma_z) \tag{22}$$

Bulk Modulus

p: Hydrostatic Pressure

$$e = -\frac{p}{k} \tag{23}$$

k: bulk modulus of the material

$$k = \frac{E}{3(1-2v)} \tag{24}$$

Shearing Strain: Modulus of Rigidity

$$\tau_{xy} = G\gamma_{xy} \tag{25}$$

$$\tau_{yz} = G\gamma yz \tag{26}$$

$$\tau_{zx} = G\gamma_{zx} \tag{27}$$

$$\frac{E}{2G} = 1 + v \tag{28}$$

Stress Concentrations

$$K = \frac{\sigma_{\text{max}}}{\sigma_{\text{avg}}} \tag{29}$$

Chapter 3 - Torsion

General

Deformation in Circular Shafts

$$\gamma = \frac{\rho\phi}{L} \tag{30}$$

$$\gamma_{max} = \frac{c\phi}{L} \tag{31}$$

$$\gamma = \frac{\rho}{c} * \gamma_{max} \tag{32}$$

Shearing Stresses in Elastic Range

$$\tau = -\frac{\rho}{c}\tau_{max} \tag{33}$$

$$\tau_{max} = \frac{Tc}{J} \tag{34}$$

$$\tau = \frac{T\rho}{J} \tag{35}$$

Polar Moment of Inertia Solid Shaft

$$J = \frac{1}{2}\pi c^4 \tag{36}$$

(18) $\mid c = \text{radius}$

Polar Moment of Inertia of a Hollow Shaft inner radius c1, outer radius c2

$$J = \frac{1}{2}\pi(c_2^4 - c_2^4) \tag{37}$$

Angle of Twist

$$\phi = \frac{TL}{JG} \tag{38}$$

$$\phi = \Sigma \frac{TL}{JG} \tag{39}$$

Statically Indeterminante Shafts

Transmission Shafts

Power P is transmitted as:

$$P = 2\pi f T \tag{40}$$

T is the torque exerted at each end of the shaft f the frequency (hz or s^{-1})

Stress Concentrations

$$\tau_{\text{max}} = K \frac{Tc}{I} \tag{41}$$

K = Stress concentration factor stress $\frac{Tc}{T}$ is computed for the smaller-diameter shaft

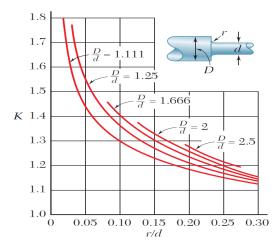


Fig. 3.28 Plot of stress concentration factors for fillets in circular shafts. (Source: W. D. Pilkey and D. F. Pilkey, Peterson's Stress Concentration Factors, 3rd ed., John Wiley & Sons, New York, 2008.)

Plastic Deformations

$$T = \int_0^c \rho \tau(2\pi d\rho) = 2\pi \int_0^c \rho^2 \tau d\rho \tag{42}$$

Modulus of Rupture

This is a ficticious value.

$$R_t = \frac{T_u c}{i} \tag{43}$$

Solid Shaft of Elastoplastic Material

Maximum Elastic Torque; Solid Circular Shaft, Radius c

$$\tau_y = \frac{1}{2}\pi c^3 \tau Y \tag{44}$$

Torque Related to ρ_y

$$T = -\frac{4}{3}T_y(1 - \frac{1}{4}\rho\rho^3 yc^3)$$
 (45)

Plastic Torque

$$T_p = \frac{4}{3}T_y \tag{46}$$

Plastic Torque Vs. Angle of Twist

$$T = \frac{4}{3}T_y(1 - \frac{1}{4}\frac{\phi^3 y}{\phi^3}) \tag{47}$$

Torsional Loading or Shaft Cross-Section Changes Along Length

$$\phi = \sum_{i} \frac{T_i L_i}{J_i G_i} \tag{48}$$

Thin-Walled Hollow Shafts

Shear Flow

$$q = \tau t \tag{49}$$

Average Shearing Stress τ at any given point in cross section

$$\tau = \frac{T}{2tA} \tag{50}$$

Chapter 4 - Pure Bending

1 - Symmetric Members in Pure Bending

$$\epsilon_x = -\frac{y}{\rho} \tag{51}$$

 ρ - Radius of curvature of the neutral surface

y - Distance from neutral surface

$\mathbf 2$ - Stresses and Deformatoins in the Elastic Range

$$\sigma_x = -\frac{y}{c}\sigma_m \tag{52}$$

c - largest distance from the neutral axis to a point in the section

Elastic Flexture formula

$$\sigma_m = \frac{Mc}{I} \tag{53}$$

$$\sigma_x = -\frac{My}{I} \tag{54}$$

Eleastic Section Modulus

$$S = \frac{I}{c} \tag{55}$$

$$\sigma_m = \frac{M}{S} \tag{56}$$

Curvature of Member

$$\frac{1}{\rho} = \frac{M}{EI} \tag{57}$$

Eccentric Axial Loading

$$\sigma_x = \frac{P}{A} - \frac{M_y}{I} \tag{58}$$

Unsymmetric Bending

$$\sigma_x = -\frac{M_z y}{I_z} + \frac{M_y z}{I_y} \tag{59}$$

General Eccentric Axial Loading

$$\sigma_x = \frac{P}{A} - \frac{M_z y}{I_z} + \frac{M_y Z}{I_y} \tag{60}$$

Curved Members

$$R = \frac{A}{\int \frac{dA}{r}} \tag{61}$$

$$\sigma_x = -\frac{My}{Ae(R-y)} \tag{62}$$

Additional Notes

Area, width, and moment of inertia for W shapes should be given on a test or found in appendix c of textbook.