

# Module 5

## Lecture 14

### Gears – Part 1



PICK  
UP YOUR  
KIT



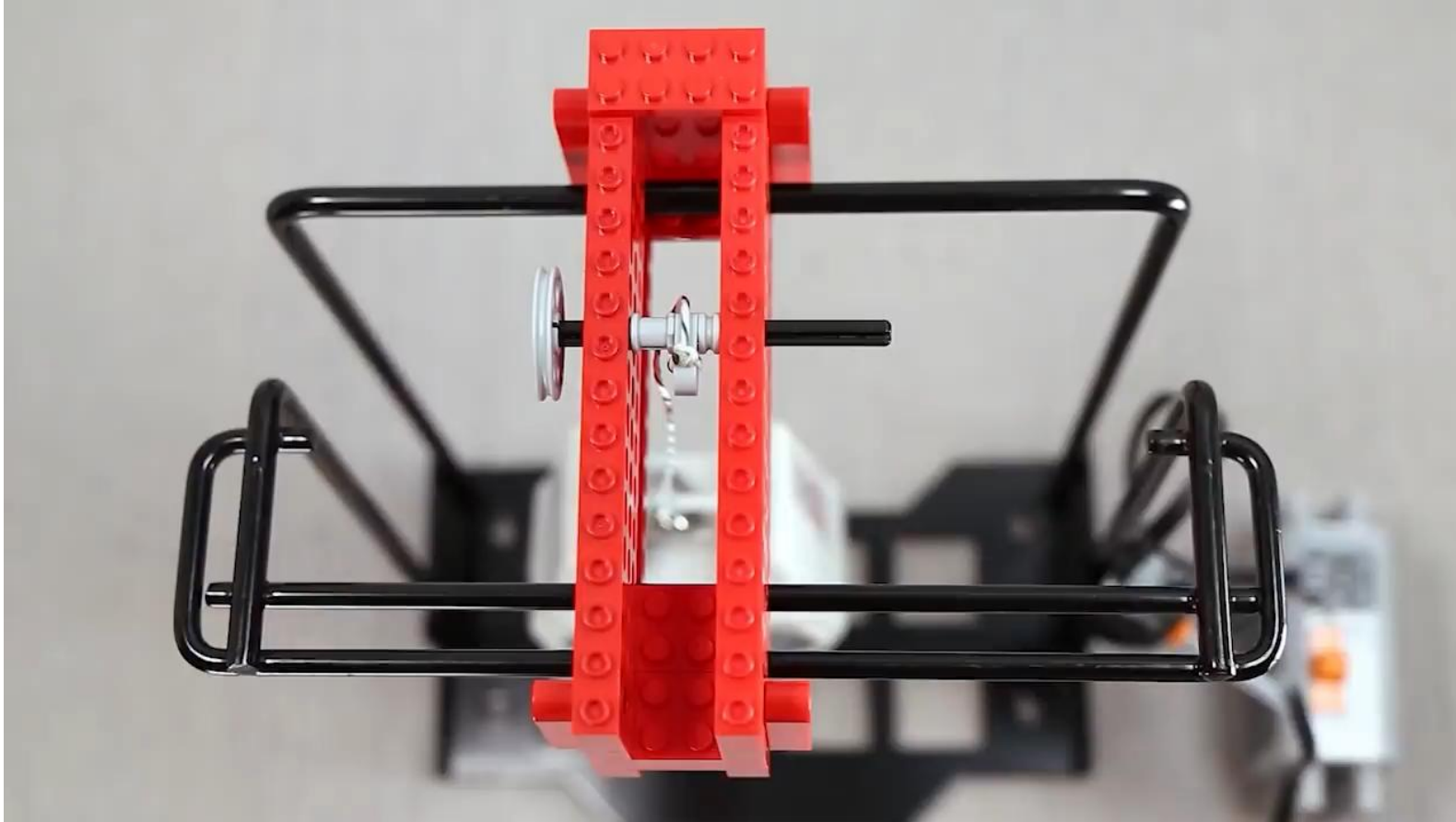
**ME 370 - Mechanical Design 1**

*“Colibri” by Derek Hugger*

\* [www.youtube.com/watch?v=1scj5sotD-E](http://www.youtube.com/watch?v=1scj5sotD-E)

# Worm gear torque, brick experiment channel

<https://www.youtube.com/watch?v=8rc-gpo3auQ>



## Practical questions while watching video:

- Why does a worm gear have a lower efficiency than a spur gear?
- Why does lubricant lead to a higher transmission of force?

# Lecture 14: Gears 1

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**Topics:** 10/13/25 Gears – Part 1 (Norton Chap 9)

## **Activities & Upcoming Deadlines**

- **HW:**
  - **HW 6 (PVA #2):** due Tuesday 10/14
  - **HW 7 (IC #1):** to be posted soon, due Tuesday 10/21
- **Lab 8: PVA analysis**
  - This is an **Individual Student** Lab. Pre-lab: READ all lab materials prior to lab time, submit pre-lab assignment
- **Project 1:**
  - Peer evaluation 2 in [CATME](#) (by 1 week after lab section for P1D5). This is a graded assignment. Provide comments if you give score(s)  $\leq 3$
- **Project 2:**
  - P2D1 (Conceptual Design Review) – **during Lab 9 (in 1 week!)**
    - Propose two possible designs for the legged dispensing robot (include positioning of the motor and battery holder). Also include theme images if possible
    - Presentation template to be posted soon
  - Parts kits will be distributed during Office Hours. CAD STEP files for parts are available on Pololu.

**Next lecture:** 10/15/25 Gears – Part 2

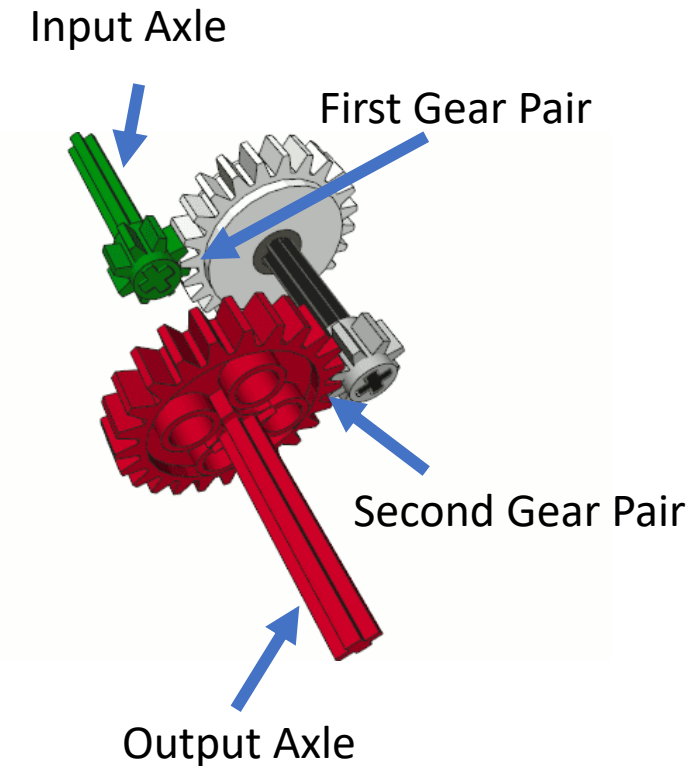
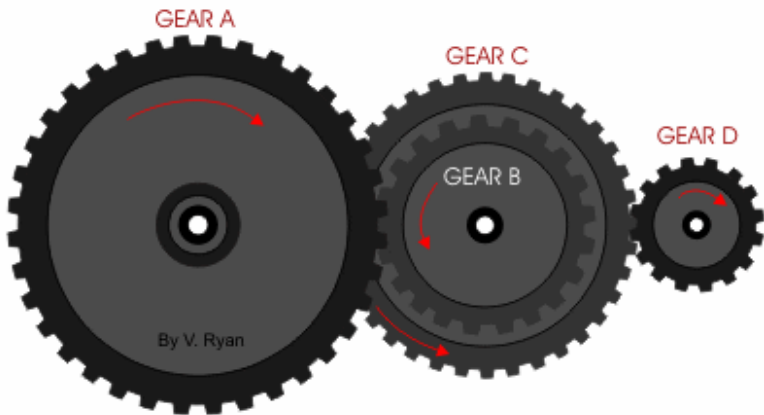
# Module 5 topics: Gears

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- Gear Uses, Types, Terminology
- Rolling cylinders – idealized gear set
- Velocity Ratio, Torque Ratio
- Power Transmission
- Gear Trains
  - Simple
  - Compound
- Pitch, Pressure Angle
- Gear Ratio
- Fundamental Law of Gearing
- Involute Tooth Shape and Nomenclature
- Normal forces in gears
- Epicyclic gear trains
  - Planetary
  - Differential

# Gears are used:

1. To transmit power
  - Either increase or decrease (a) torque or (b) speed of rotation, not both
2. To move rotational motion to a different axis or direction
3. To keep the rotation of two axes synchronized



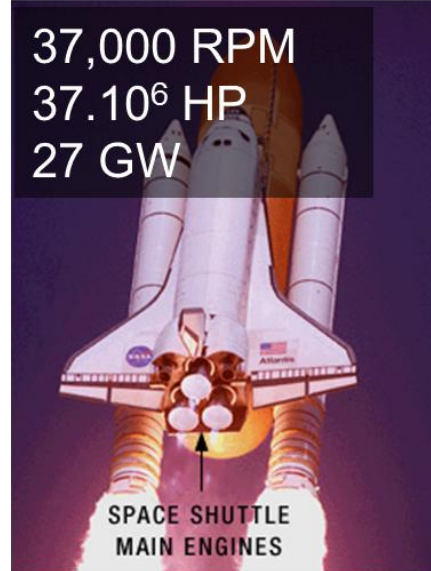
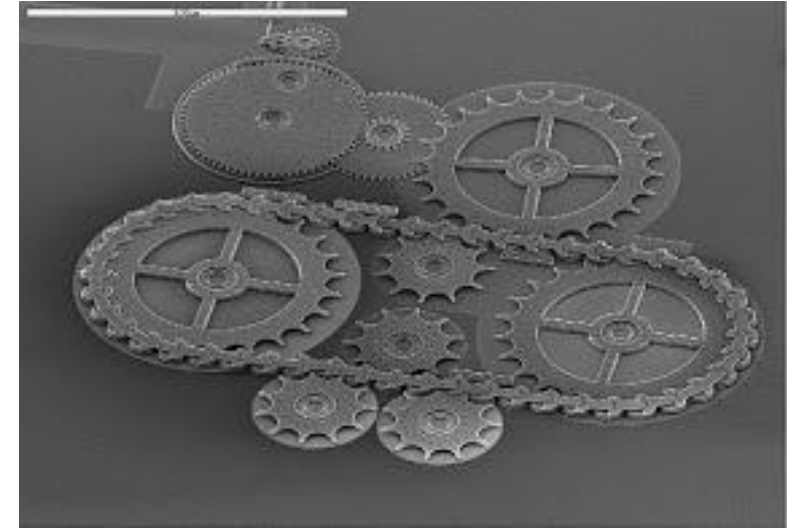


# Transmission of power:



## Small:

World's smallest gear chain drive by Sandia National Lab. Chain sprockets separated by ~200 microns.



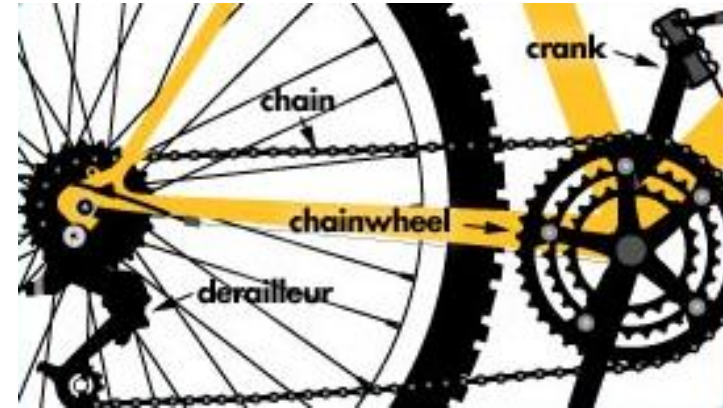
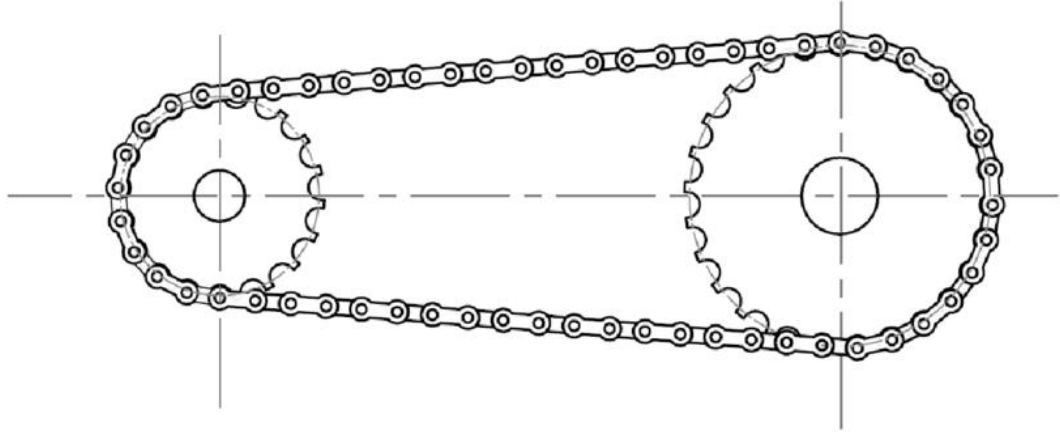
## Large:

Large gears can be made to over 100 feet in diameter.

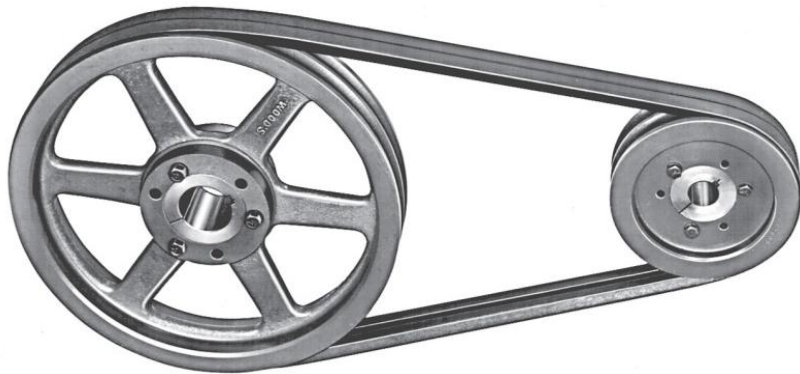


# Belts and chains – when axes are separated

Chain drive:



Belt drive:



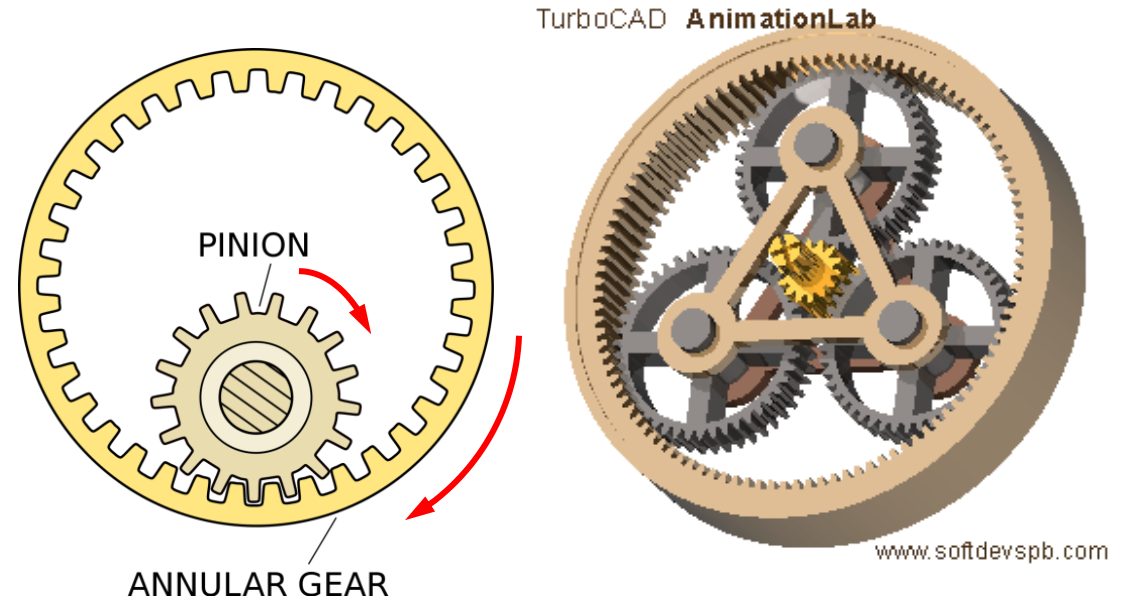


# External vs. internal gear trains

External gear train:  
Motion of gears is opposite



Internal gear train:  
Motion of gears is the same





# Common Gear Types

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Spur



Bevel Spur



Worm



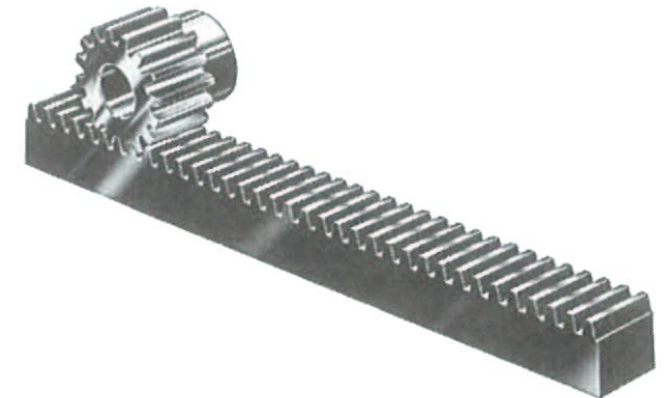
Helical



Helical Bevel



Rack-pinion



# Special gears: helical and beveled

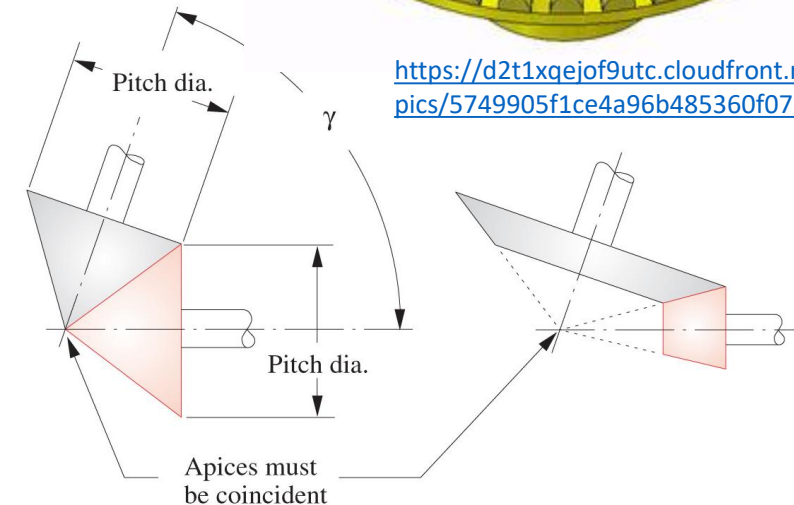
## Helical Gearset

- Quieter and smoother than spur gears
- Can handle larger loads
- Less efficient than spur  
 $\text{Efficiency} = \text{Power}_{\text{out}} / \text{Power}_{\text{in}}$
- Can transmit non-parallel axes



## Beveled Gearset

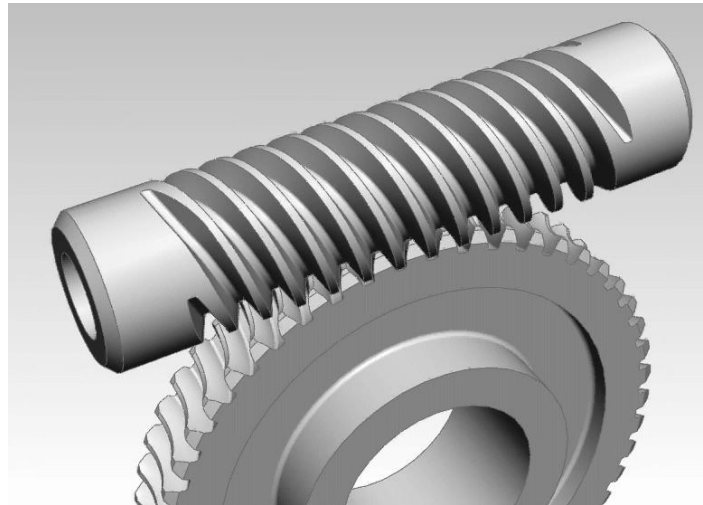
- Transmission between non-parallel axes. Angle can be different than  $90^\circ$
- Based on rolling cones



# Special gears: worm and rack & pinion

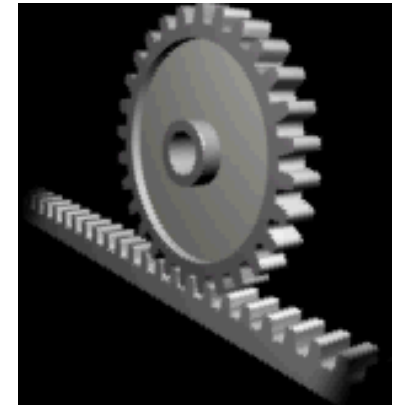
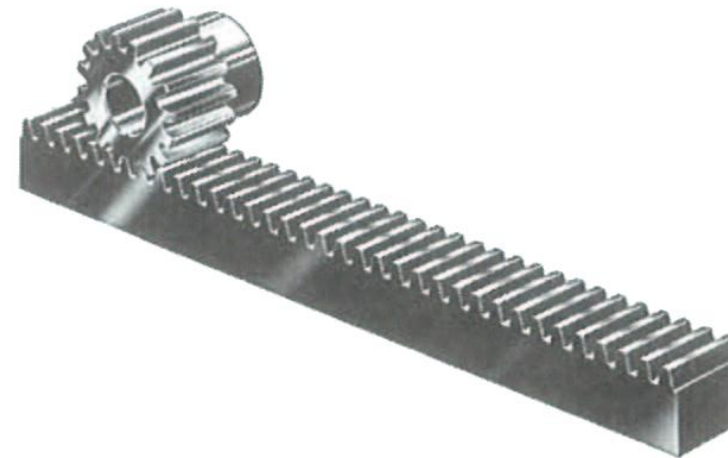
## Worm gearset

- **Worm:** on top is similar to a screw thread
- **Worm gear:** similar to a spur gear
- *A high gear ratio in small packaging*
- Self-locking (*not back-drivable*)



## Rack and pinion

- **Rack:** infinite gear radius
- **Pinion:** spur gear
- *Converts linear to and from rotary motion*



[https://en.m.wikipedia.org/wiki/Rack\\_and\\_pinion#/media/File%3ARack\\_and\\_pinion\\_animation.gif](https://en.m.wikipedia.org/wiki/Rack_and_pinion#/media/File%3ARack_and_pinion_animation.gif)

[https://upload.wikimedia.org/wikipedia/commons/c/c3/Worm\\_Gear.gif](https://upload.wikimedia.org/wikipedia/commons/c/c3/Worm_Gear.gif)

# Toothed Gears

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The gear teeth have three advantages compared to friction (belt) drives:

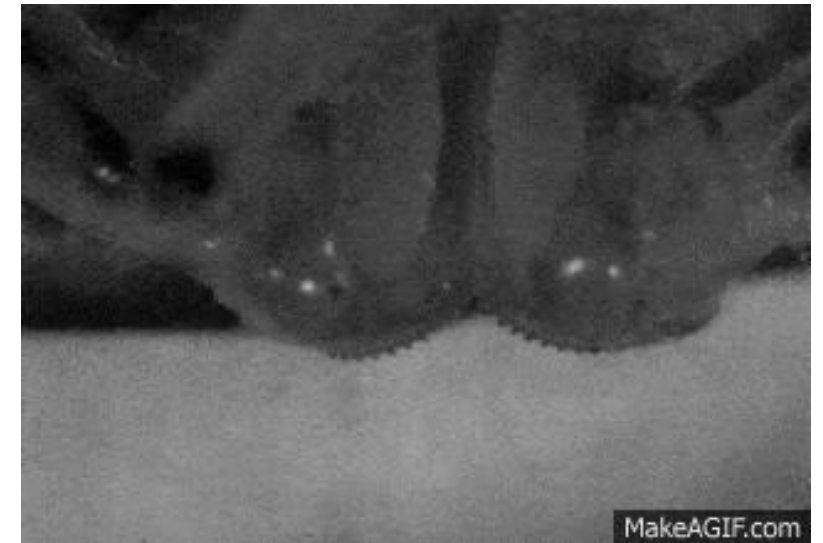
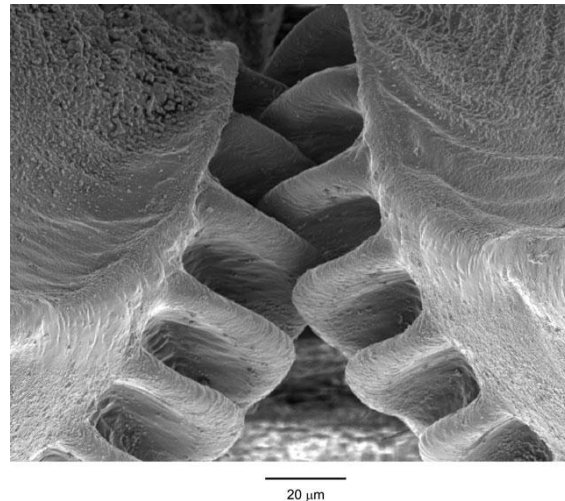
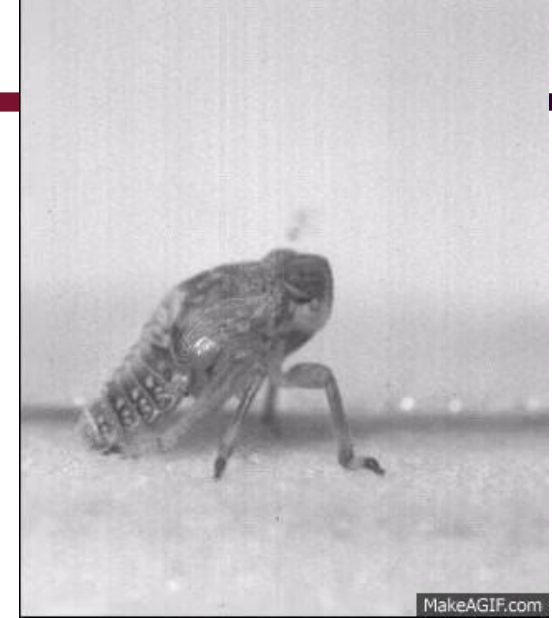
- 1) Prevents slippage between the gears.
- 2) Makes it possible to determine exact gear ratios.  
**Count number of teeth and divide,  $N_3 / N_2$ .**
- 3) Slight imperfections in actual diameter and circumference of the two gears do not matter.

**Gear ratio is controlled by number of teeth even if diameters are a bit off.**



# Toothed Gears in Nature

- *Issus* nymph: Adolescent plant hopper insect and one of the fastest accelerators in the animal kingdom
- To synchronize their legs for power jump
- Seen only in “babies” where nervous system is not ready to handle this kind of synchronization
- Absent in adults perhaps not to hinder its jumping ability if hurt

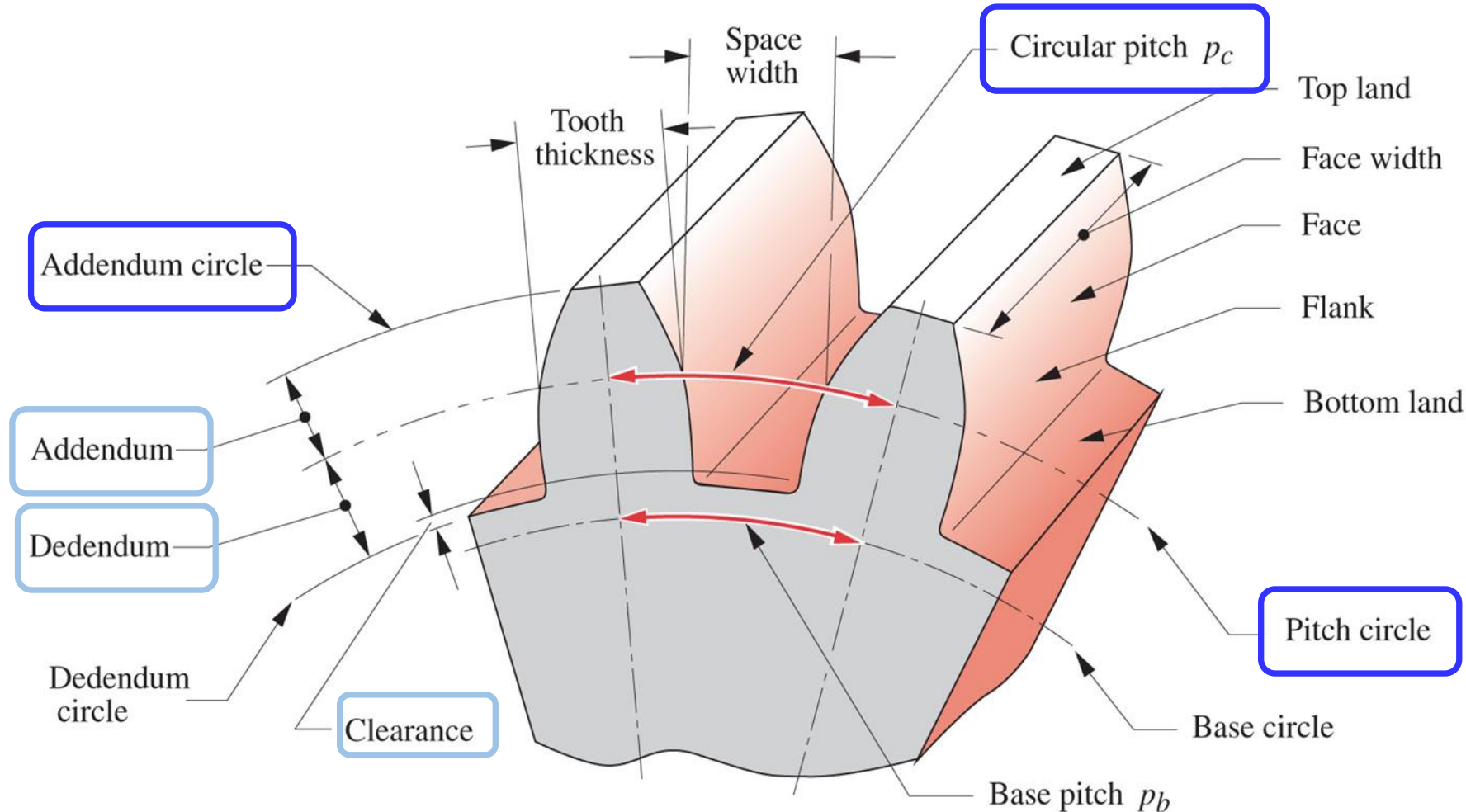


# Parts of a Gear Tooth

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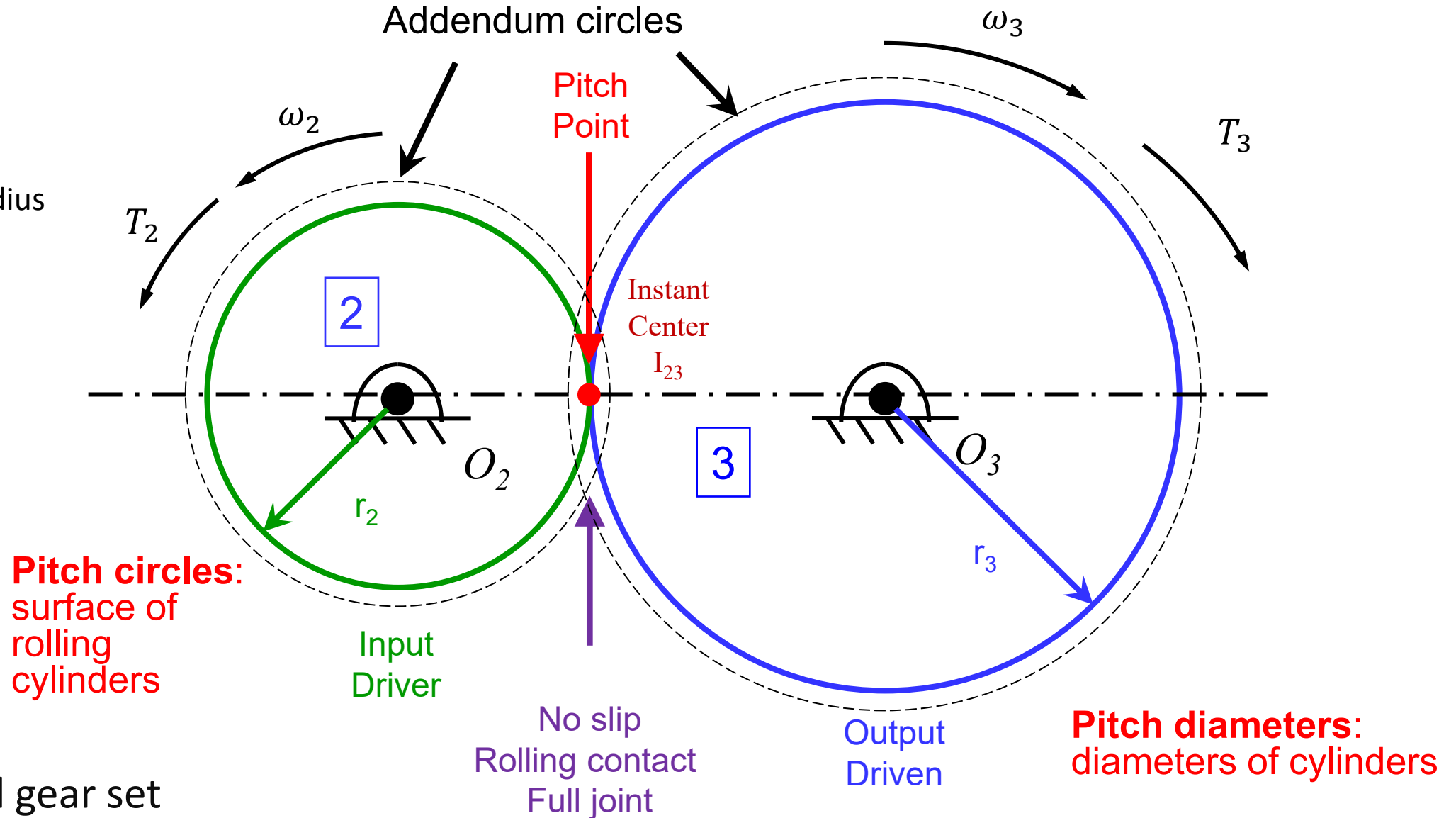
$$p_c = \frac{\pi d}{N}$$

Where  $d$  = pitch diameter  
 $N$  = # teeth

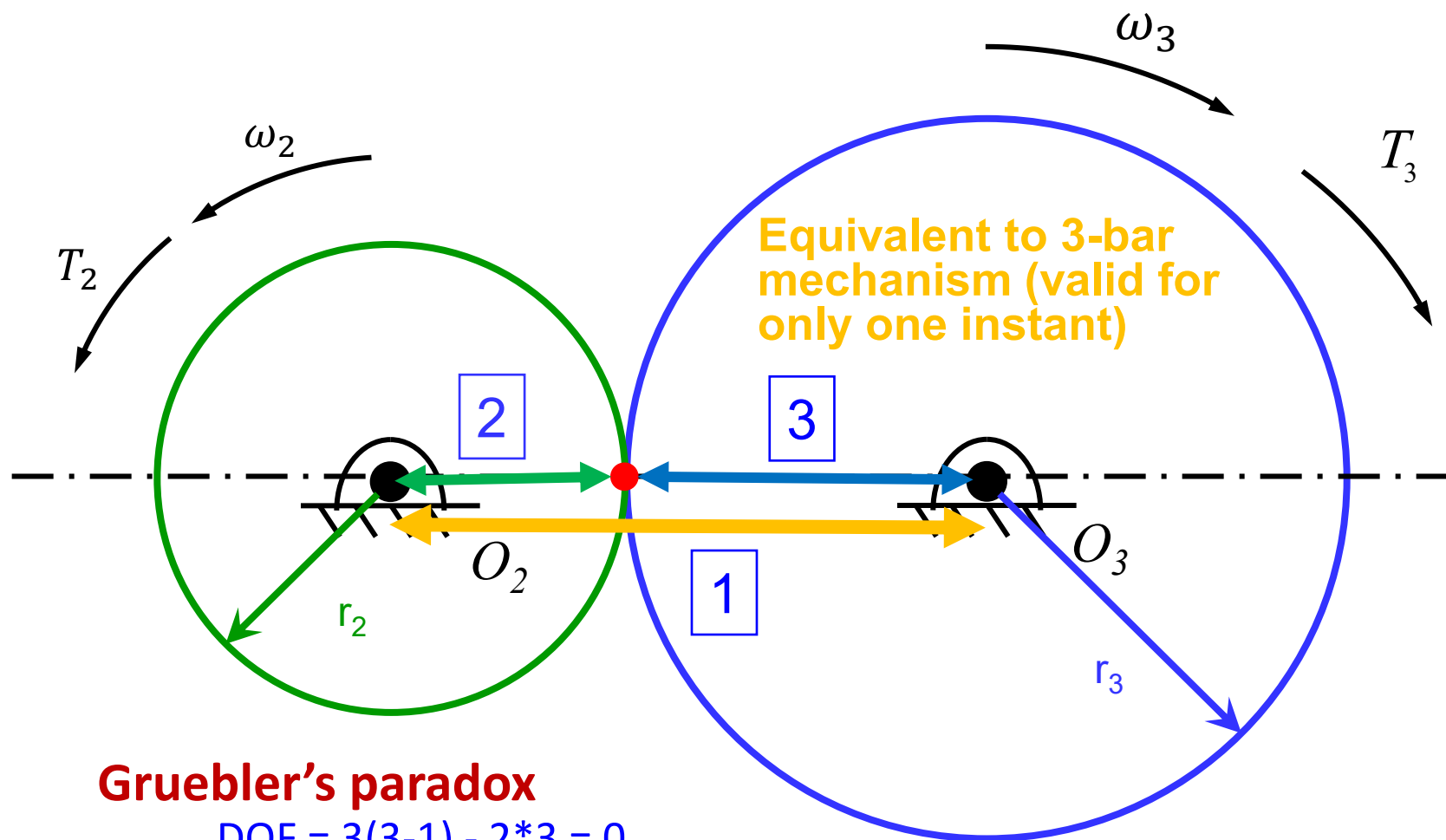


# Rolling cylinders (idealized toothed gear set or friction drive)

**Key terms:**  
Pitch circle  
Pitch diameter/radius  
Pitch point



# Recall Gruebler's paradox



## Gruebler's paradox

$$\text{DOF} = 3(3-1) - 2 \cdot 3 = 0$$

Gruebler's predicts no motion

Ground length = sum of two radii

Gruebler's equation does not account for link size or shape.

Moral : Watch out for higher symmetry (e.g., parallel links, summed length)

An external gear set



# Velocity ratio: Modeling gears as rolling cylinders:

Surface **velocity** at the contact point, or **pitch point**, is the **same**:

Given:

$$r_{in} = 4 \text{ cm} \quad r_{out} = 6 \text{ cm} \quad \omega_{in} = 90 \text{ rpm}$$

$$\omega_{out} = ? = \frac{-r_{in}}{r_{out}} \omega_{in} = \frac{-4}{6} \cdot 90 \text{ rpm} =$$

$$\mathbf{-60 \text{ rpm}}$$

Angular velocity ratio ( $m_V$ ):

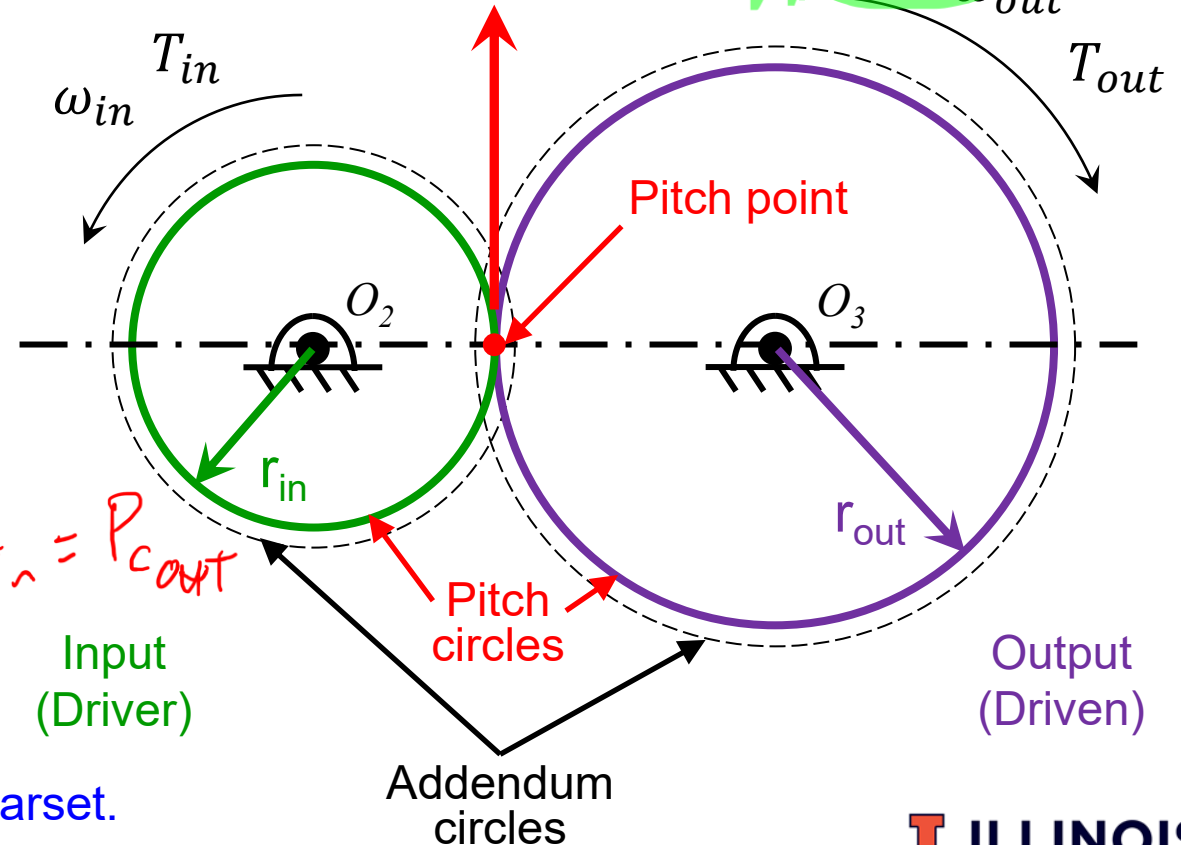
$$\therefore m_V = \pm \frac{\omega_{out}}{\omega_{in}} = \pm \frac{r_{in}}{r_{out}} = \pm \frac{d_{in}}{d_{out}} = \pm \frac{N_{in}}{N_{out}}$$

Negative sign if external gearset, Positive sign if internal gearset.

$$V_p = V_{in} = V_{out}; \text{ recall: } V = r\omega$$

$$V_p = V_{in} = r_{in}\omega_{in} = V_{out} = -r_{out}\omega_{out}$$

External gear set  
negative sign



# Torque Ratio ( $m_T$ )

Tangential *force* at the contact point, or *pitch point*, is the **same**:

$$r_{in} = 4 \text{ cm} \quad r_{out} = 6 \text{ cm} \quad T_{in} = 1 \text{ Nm}$$

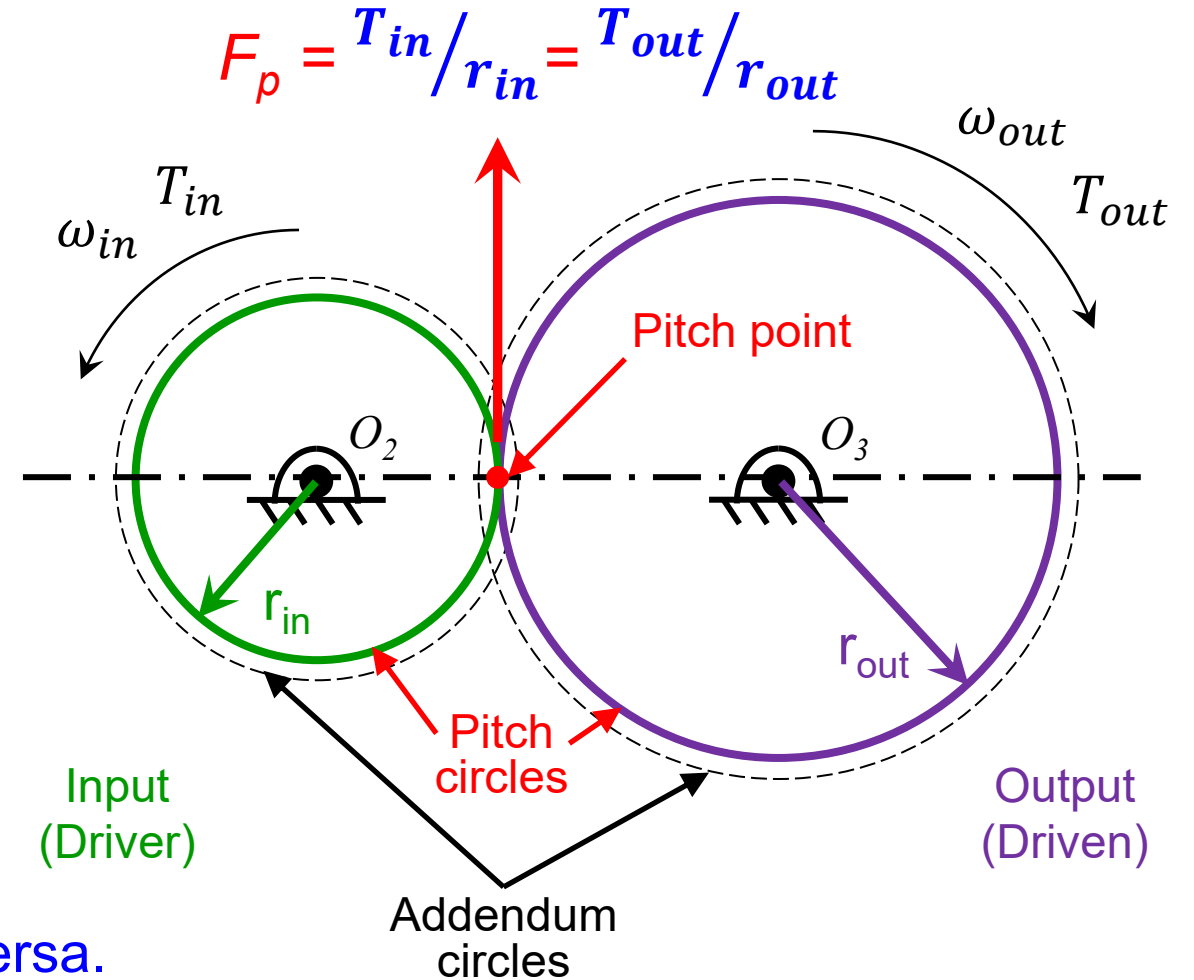
$$T_{out} = ? = \frac{r_{out}}{r_{in}} T_{in} = \frac{6}{4} \cdot 1 \text{ Nm} =$$

**1.5 Nm**

Torque ratio ( $m_T$ ):

$$\therefore m_T = \frac{T_{out}}{T_{in}} = \frac{\omega_{in}}{\omega_{out}} = \frac{1}{m_V}$$

Gearsets exchange torque for velocity or vice versa.



# Power transmission

- Assuming a gearset is highly efficient, then  $P_{out} = \sim P_{in}$

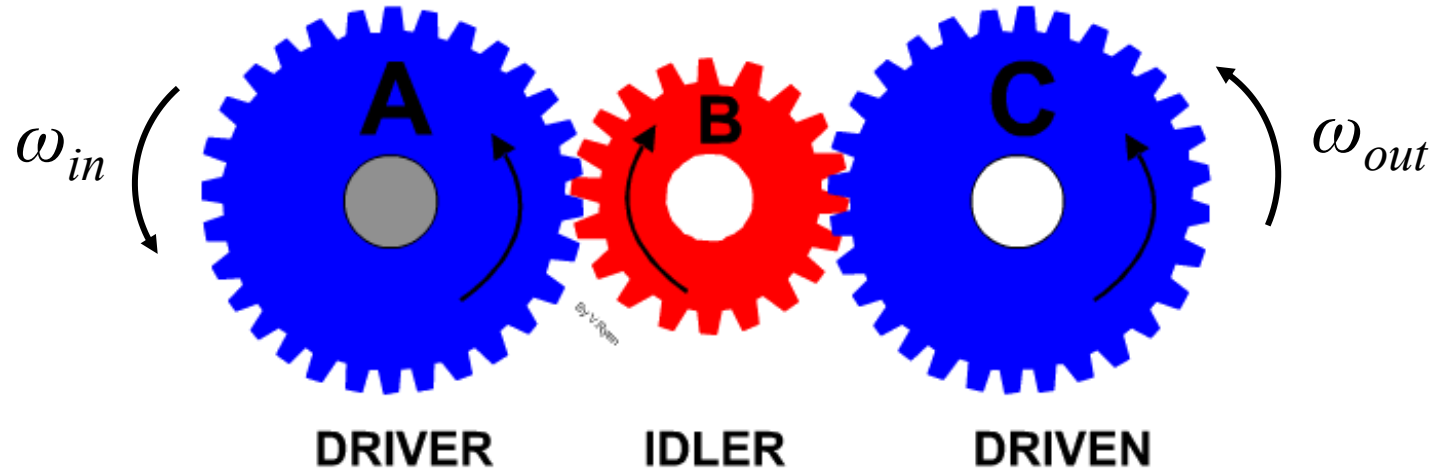
$$P = \vec{T} \cdot \vec{\omega} = \vec{T}_{in} \cdot \vec{\omega}_{in} = \sim \vec{T}_{out} \cdot \vec{\omega}_{out}$$

$$|m_V| \equiv \frac{\omega_{out}}{\omega_{in}} = \pm \frac{r_{in}}{r_{out}} = \pm \frac{d_{in}}{d_{out}} = \pm \frac{N_{in}}{N_{out}} = \frac{1}{|m_T|} \equiv \frac{T_{in}}{T_{out}}$$

Velocity Ratio  
Gear Ratio

Torque Ratio

# Simple gear train



<https://technologystudent.com/gears1/gears2.htm>

- Middle gear is called an **idler** gear.
  - Has no contribution to gear ratio.
  - Only changes direction of input to output.
  - Odd # gears: input & output same direction
  - Even # gears: input & output opposite direction
- Typically, do not use more than one idler gear. If want to cover longer distances, then use chain or belt.

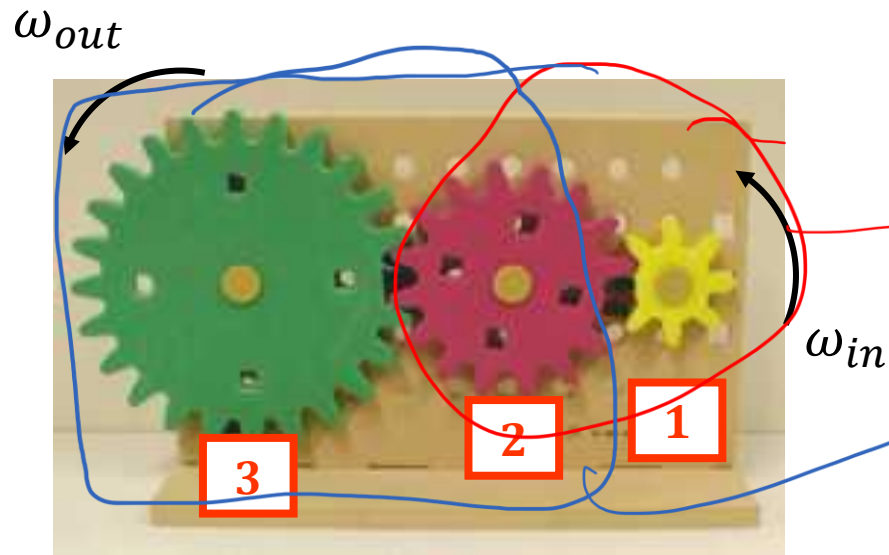
All gears on different shafts



<https://quizlet.com/255781174/simple-gear-train-with-idler-diagram/>



# Simple gear train



$$|m_V| \equiv \frac{\omega_{out}}{\omega_{in}} = \pm \frac{r_{in}}{r_{out}} = \pm \frac{d_{in}}{d_{out}} = \pm \frac{N_{in}}{N_{out}} = \frac{1}{|m_T|} \equiv \frac{T_{in}}{T_{out}}$$

$$m_V \equiv \frac{\omega_{out}}{\omega_{in}} = \left( -\frac{N_1}{N_2} \right) \left( -\frac{N_2}{N_3} \right) = \left( \frac{N_1}{N_3} \right) = \left( \frac{8}{24} \right) = \frac{1}{3}$$

Negative because external gear trains

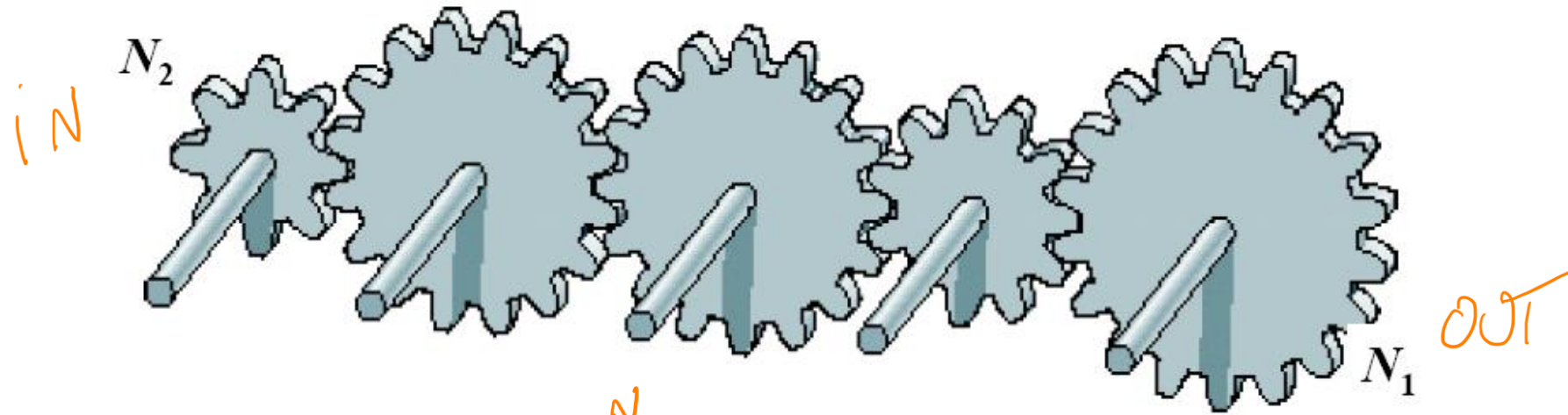
$$m_T \equiv \frac{T_{out}}{T_{in}} = \frac{1}{m_V} = 3$$

$$N_3 = 24 \quad N_2 = 16 \quad N_1 = 8$$

$$\omega_{in} = 90 \text{ rpm} \quad \omega_{out} = 30 \text{ rpm}$$

$$T_{in} = 1 \text{ Nm} \quad T_{out} = 3 \text{ Nm}$$

# Longer simple gear trains



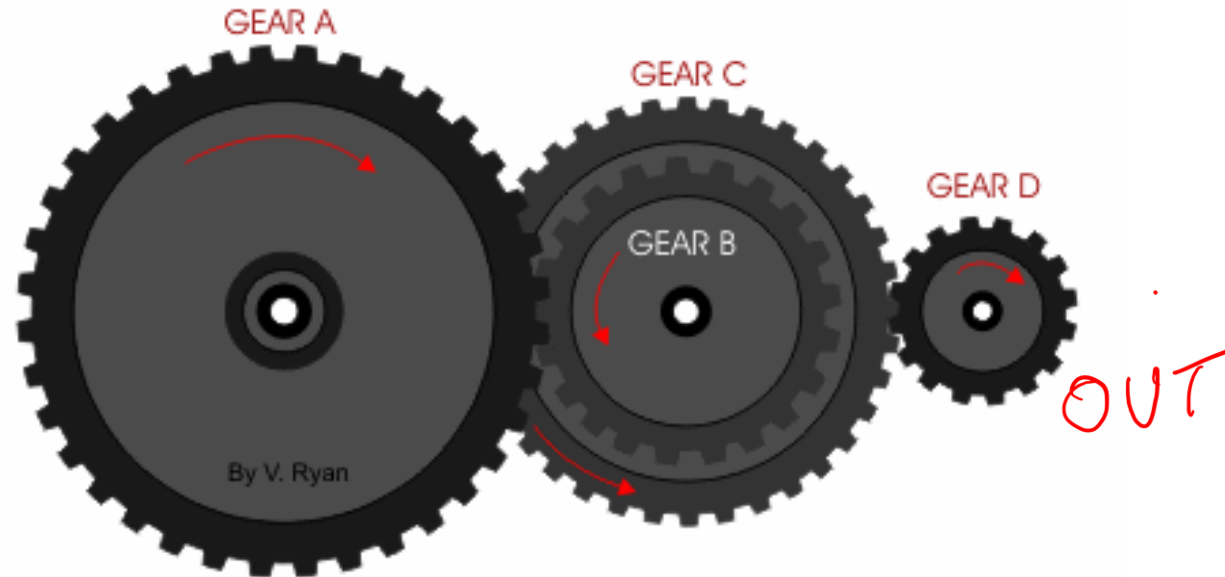
$$\frac{\omega_1}{\omega_2} = - \frac{N_2}{N_1}$$

Even number of gears

$$\frac{\omega_1}{\omega_2} = \frac{N_2}{N_1}$$

Odd number of gears

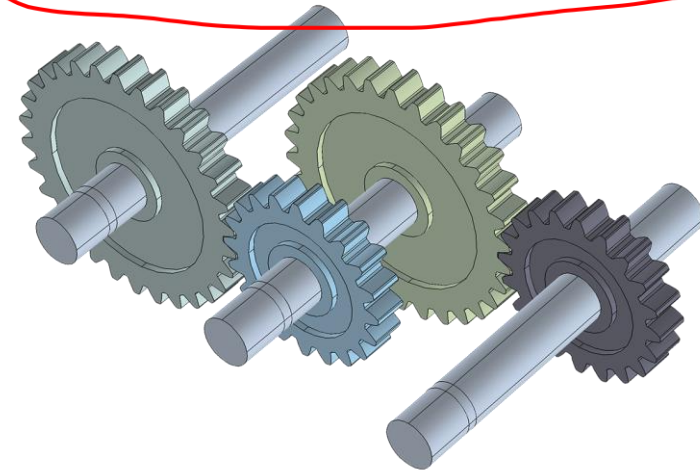
# Compound gear trains



<http://www.technologystudent.com/gears1/gears8.htm>

A & C are **drivers** (in)  
B & D are **driven** (out)

Some gears share same shaft



<https://images.app.goo.gl/PwfqXyj7cR33pdzF8>

# Compound gear trains

$$m_V = \pm \frac{\prod(N_{in \text{ or "driver gears"}})}{\prod(N_{out \text{ or "driven gears"}})}$$

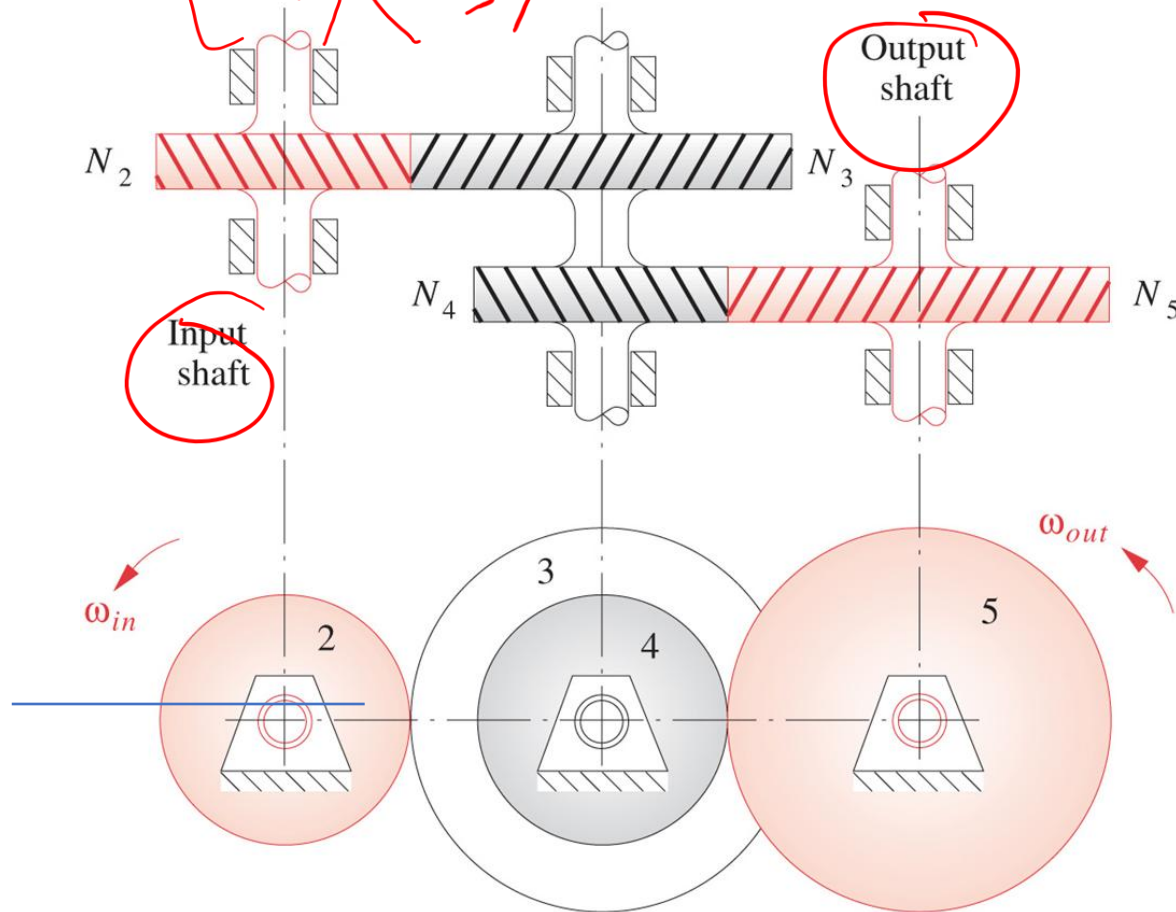
$$= \frac{N_2 N_4}{N_3 N_5} = + \frac{100 \times 100}{150 \times 150} = + 0.44$$

Some gears share the same shaft:

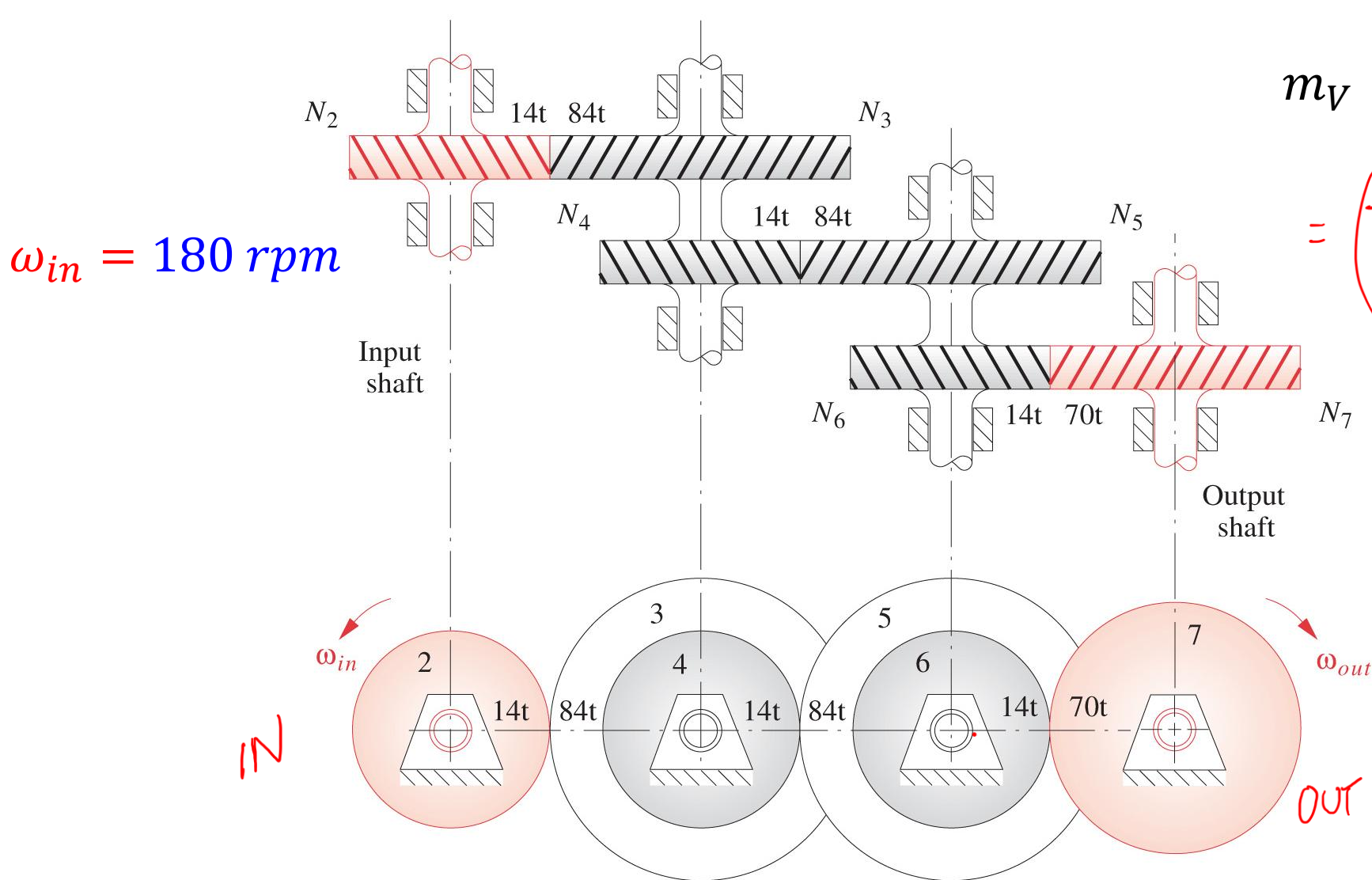
2 & 4 are drivers (*in*)  
3 & 5 are driven (*out*)

$$N_2 = N_4 = 100$$

$$N_3 = N_5 = 150$$



# What is the output rpm?



$$m_V = \pm \frac{\prod (N_{\text{driver gears}})}{\prod (N_{\text{driven gears}})}$$

$$= \left( -\frac{N_2}{N_3} \right) \left( -\frac{N_4}{N_5} \right) \left( -\frac{N_6}{N_7} \right)$$

$$= -0.0056$$

$$m_V \equiv \frac{\omega_{out}}{\omega_{in}}$$

$$\omega_{out} = -0.0056 \times 180$$

$$= -1 \text{ rpm}$$