

Module 5

Lecture 14

Gears – Part 1



PICK
UP YOUR
KIT



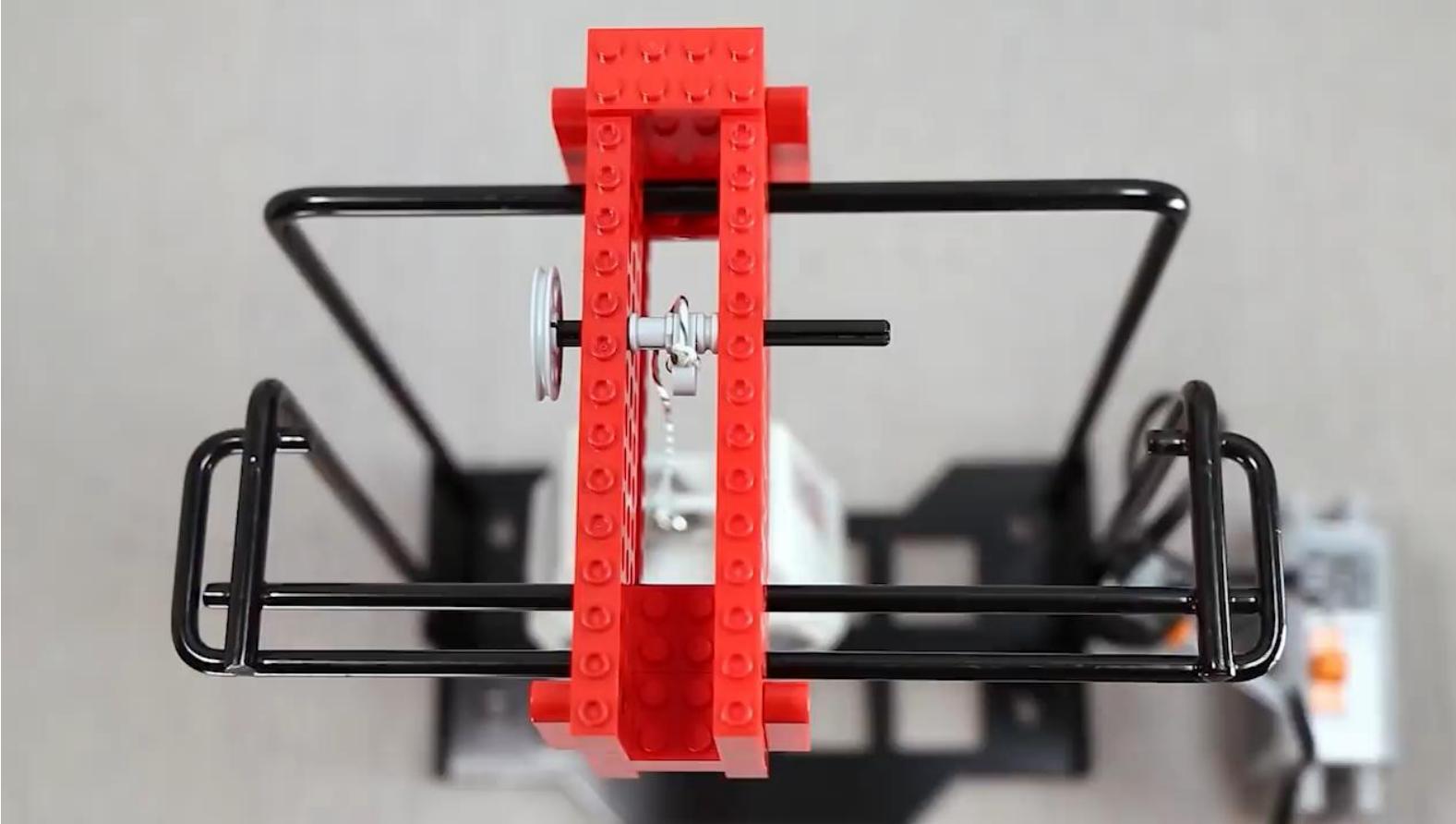
ME 370 - Mechanical Design 1

"Colibri" by Derek Hugger

* www.youtube.com/watch?v=1scj5sotD-E

Worm gear torque, brick experiment channel

<https://www.youtube.com/watch?v=8rc-gpo3auQ>



Practical questions while watching video:

- Why does a worm gear have a lower efficiency than a spur gear?
- Why does lubricant lead to a higher transmission of force?

Lecture 14: Gears 1

Topics: 10/13/25 Gears – Part 1 (Norton Chap 9)

Activities & Upcoming Deadlines

- **HW:**

- **HW 6 (PVA #2):** due Tuesday 10/14
- **HW 7 (IC #1):** to be posted soon, due Tuesday 10/21

- **Lab 8: PVA analysis**

- This is an **Individual Student Lab**. Pre-lab: READ all lab materials prior to lab time, submit pre-lab assignment

- **Project 1:**

- Peer evaluation 2 in [CATME](#) (by 1 week after lab section for P1D5). This is a graded assignment. Provide comments if you give score(s) ≤ 3

- **Project 2:**

- P2D1 (Conceptual Design Review) – **during Lab 9 (in 1 week!)**

- Propose two possible designs for the legged dispensing robot (include positioning of the motor and battery holder). Also include theme images if possible
- Presentation template to be posted soon

- Parts kits will be distributed during Office Hours. CAD STEP files for parts are available on Pololu.

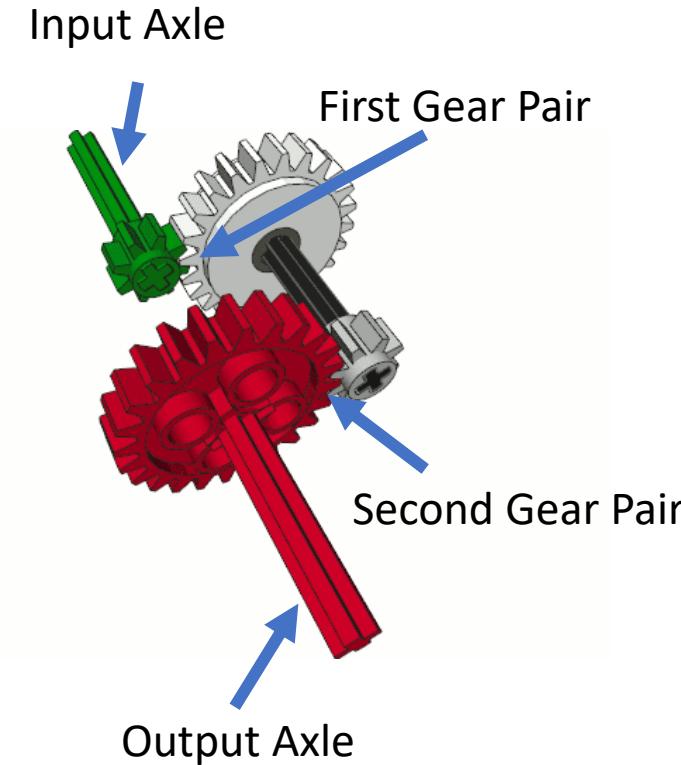
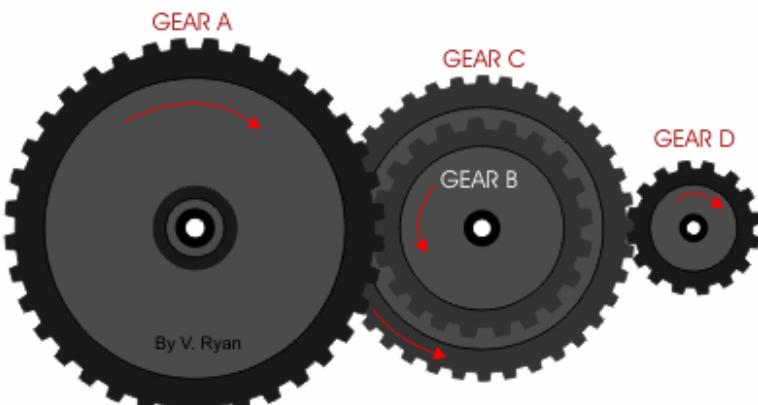
Next lecture: 10/15/25 Gears – Part 2

Module 5 topics: Gears

- Gear Uses, Types, Terminology
- Rolling cylinders – idealized gear set
- Velocity Ratio, Torque Ratio
- Power Transmission
- Gear Trains
 - Simple
 - Compound
- Pitch, Pressure Angle
- Gear Ratio
- Fundamental Law of Gearing
- Involute Tooth Shape and Nomenclature
- Normal forces in gears
- Epicyclic gear trains
 - Planetary
 - Differential

Gears are used:

1. To transmit power
 - Either increase or decrease (a) torque or (b) speed of rotation, not both
2. To move rotational motion to a different axis or direction
3. To keep the rotation of two axes synchronized

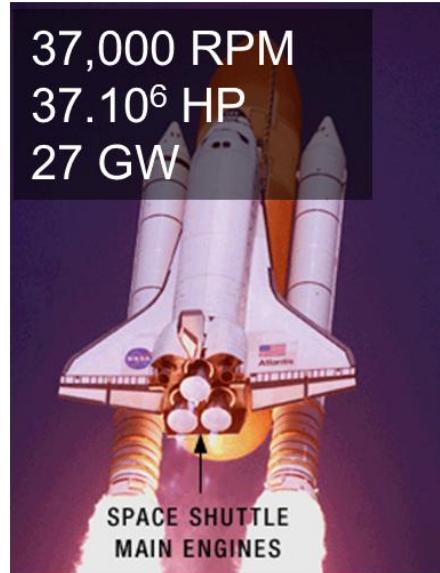


Transmission of power:



Small:

World's smallest gear chain drive by Sandia National Lab. Chain sprockets separated by ~200 microns.



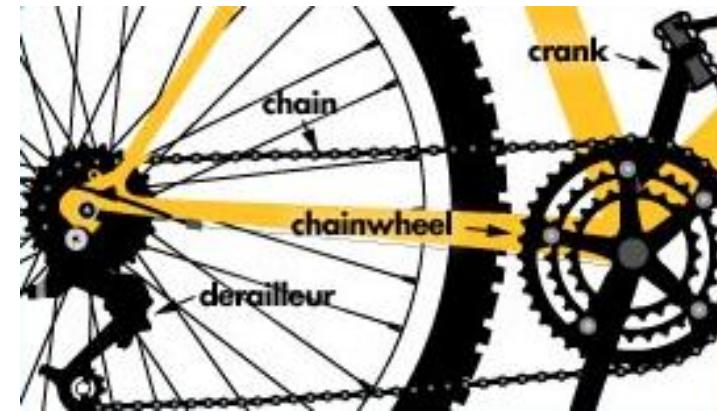
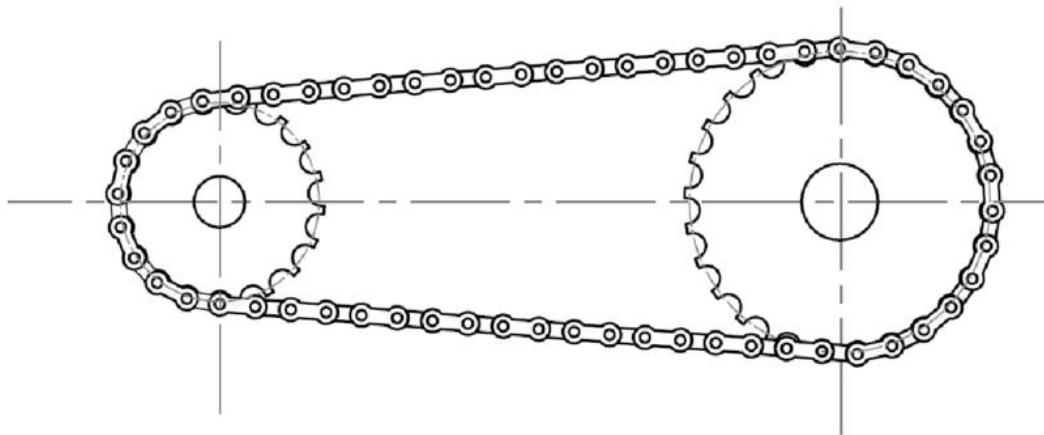
Large:

Large gears can be made to over 100 feet in diameter.

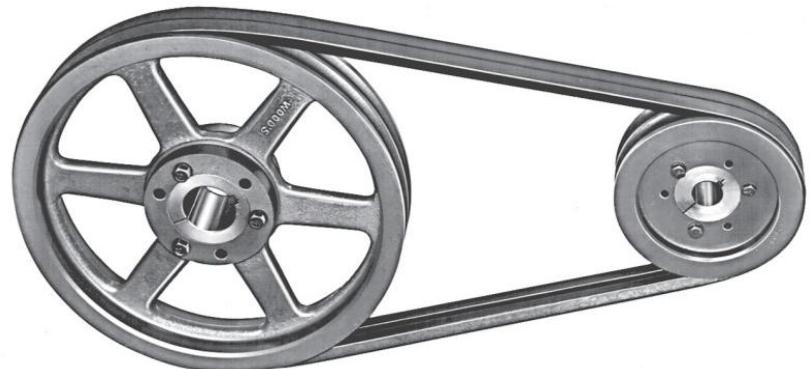


Belts and chains – when axes are separated

Chain drive:



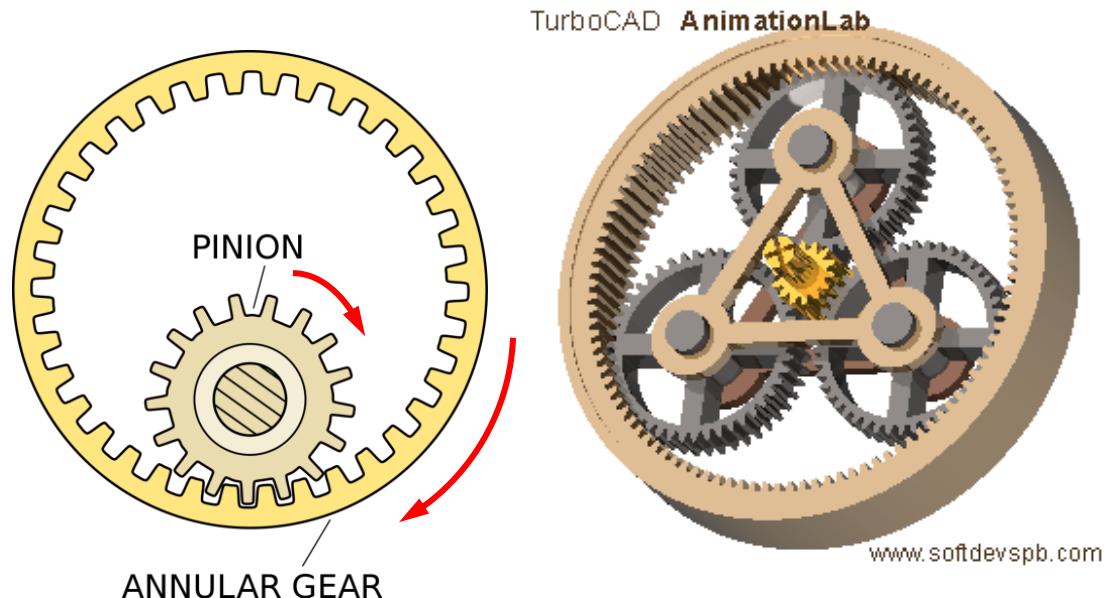
Belt drive:



External vs. internal gear trains



Internal gear train:
Motion of gears is the same



Common Gear Types

Spur



Bevel Spur



Worm



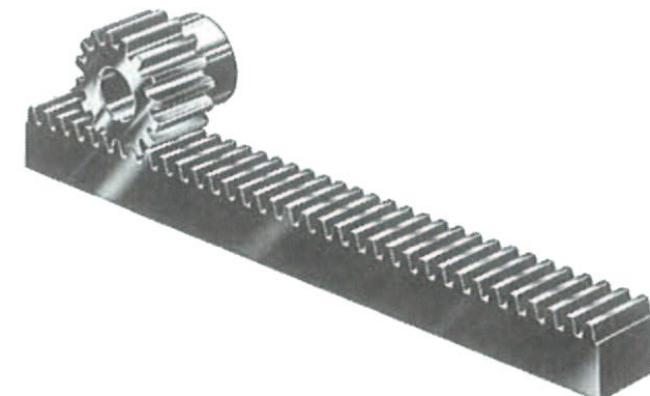
Helical



Helical Bevel



Rack-pinion



Special gears: helical and beveled

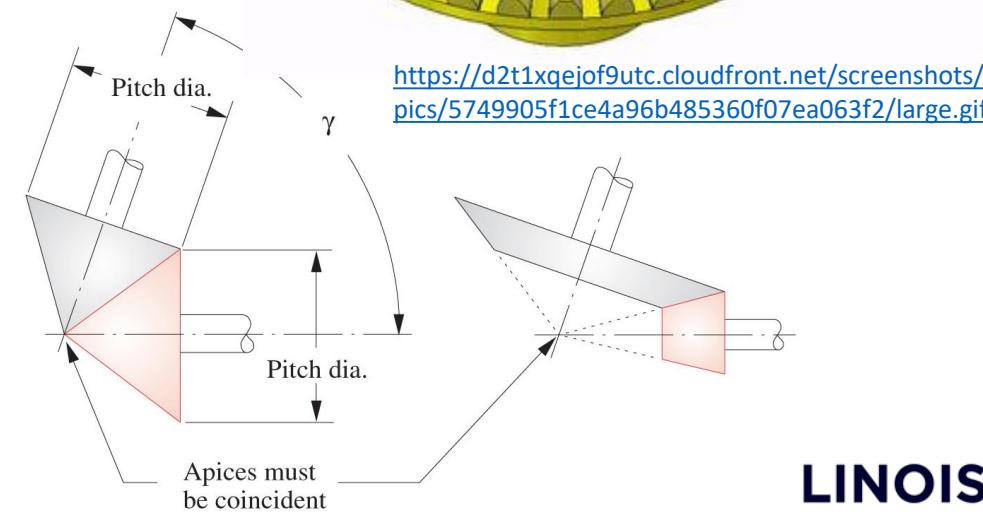
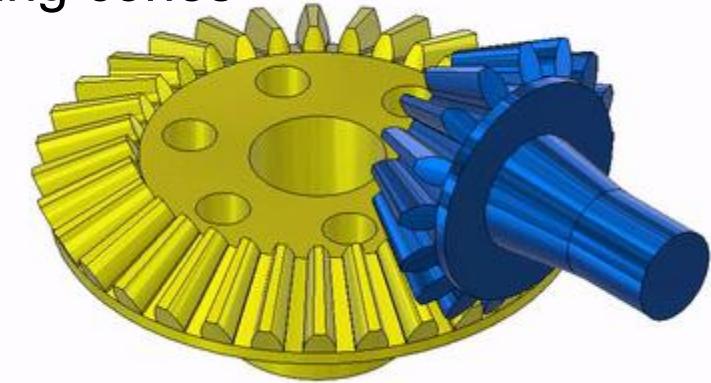
Helical Gearset

- Quieter and smoother than spur gears
- Can handle larger loads
- Less efficient than spur
Efficiency = $\text{Power}_{\text{out}} / \text{Power}_{\text{in}}$
- Can transmit non-parallel axes



Beveled Gearset

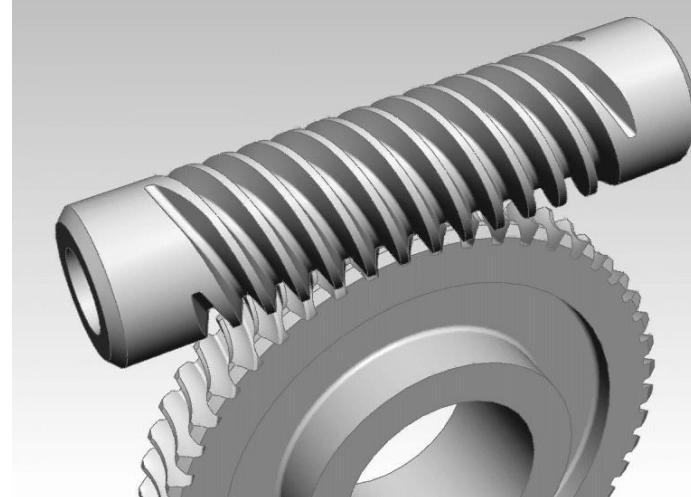
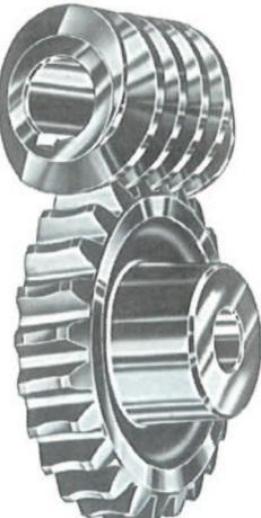
- Transmission between non-parallel axes. Angle can be different than 90°
- Based on rolling cones



Special gears: worm and rack & pinion

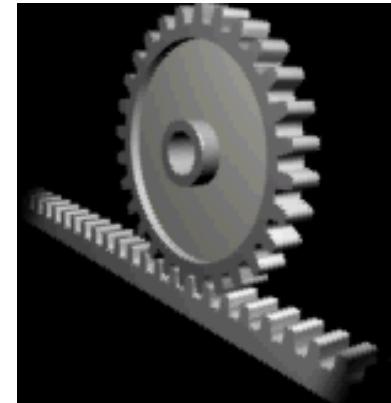
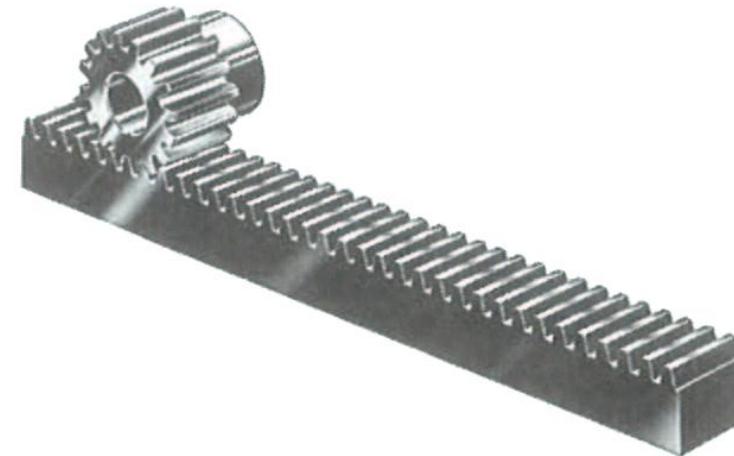
Worm gearset

- **Worm:** on top is similar to a screw thread
- **Worm gear:** similar to a spur gear
- *A high gear ratio in small packaging*
- Self-locking (*not back-drivable*)



Rack and pinion

- **Rack:** infinite gear radius
- **Pinion:** spur gear
- *Converts linear to and from rotary motion*



https://en.m.wikipedia.org/wiki/Rack_and_pinion#/media/File%3ARack_and_pinion_animation.gif

https://upload.wikimedia.org/wikipedia/commons/c/c3/Worm_Gear.gif

Toothed Gears

The gear teeth have three advantages compared to friction (belt) drives:

- 1)** Prevents slippage between the gears.
- 2)** Makes it possible to determine exact gear ratios.

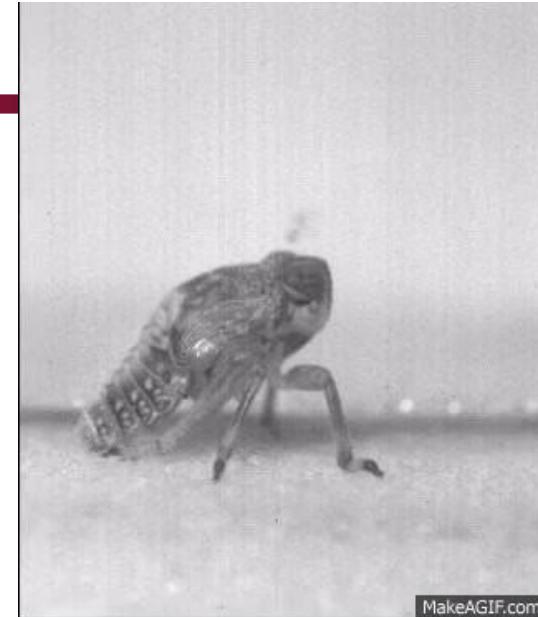
Count number of teeth and divide, N_3 / N_2 .

- 3)** Slight imperfections in actual diameter and circumference of the two gears do not matter.

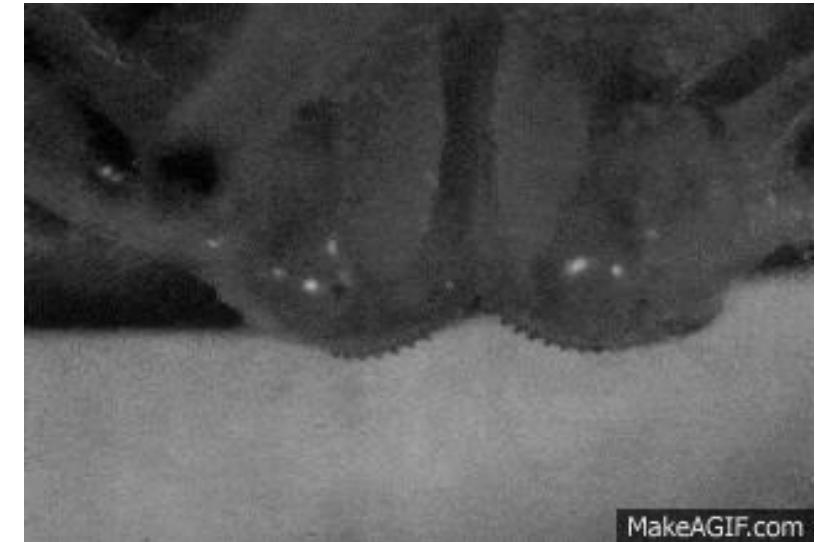
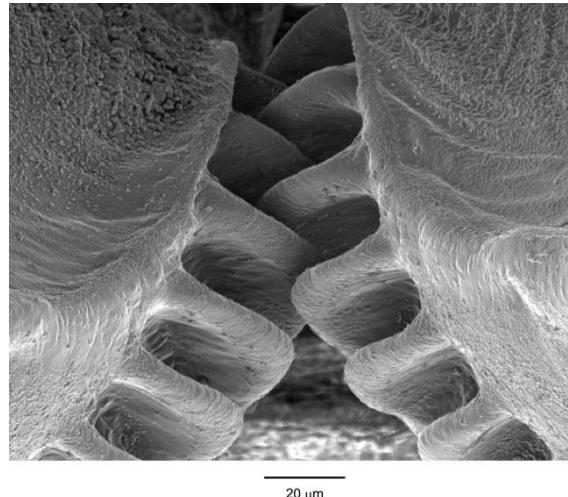
Gear ratio is controlled by number of teeth even if diameters are a bit off.

Toothed Gears in Nature

- *Issus nymph*: Adolescent plant hopper insect and one of the fastest accelerators in the animal kingdom
- To synchronize their legs for power jump
- Seen only in “babies” where nervous system is not ready to handle this kind of synchronization
- Absent in adults perhaps not to hinder its jumping ability if hurt



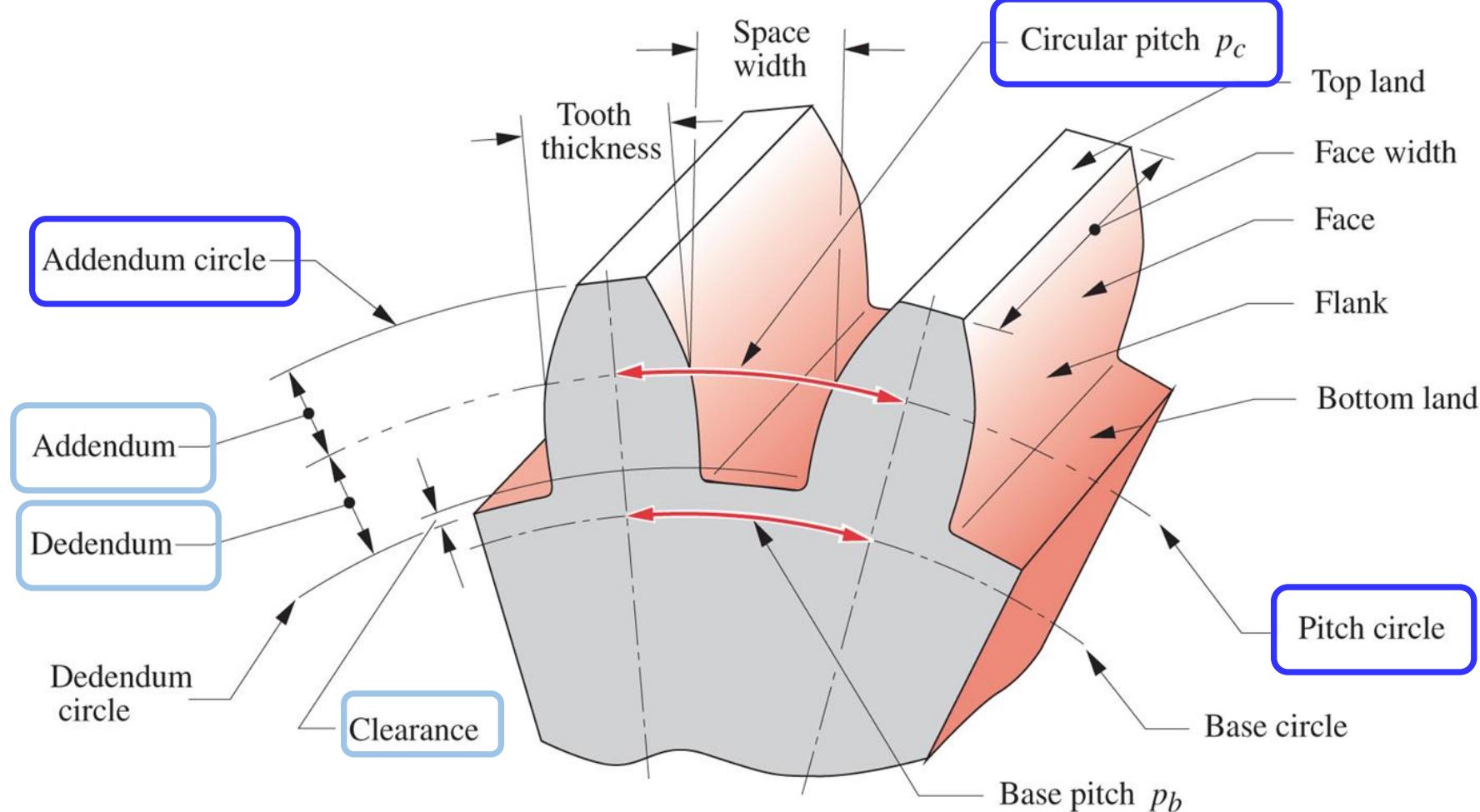
MakeAGIF.com



MakeAGIF.com

Parts of a Gear Tooth

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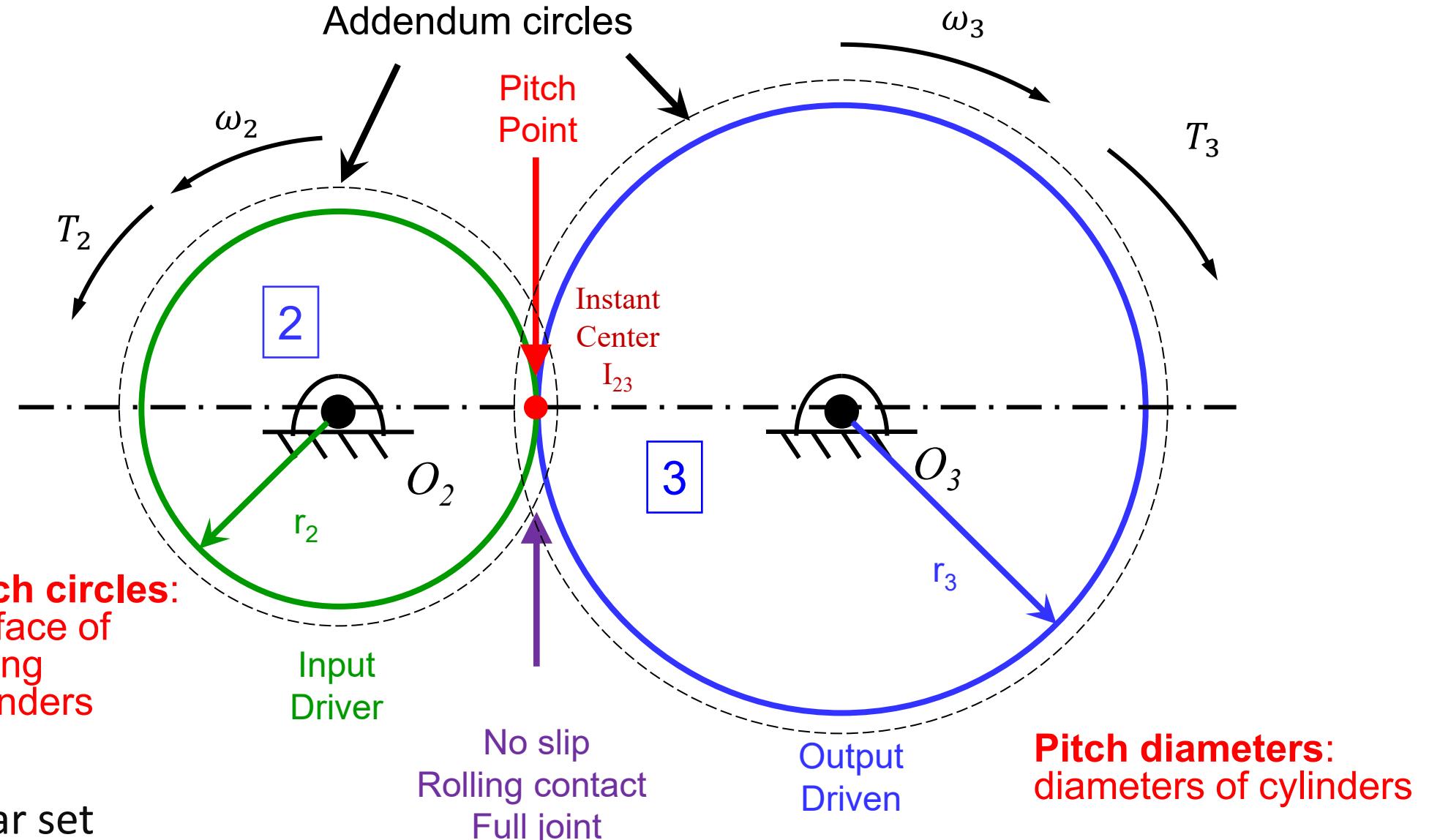
$$p_c = \frac{\pi d}{N}$$

Where d =pitch diameter
 N =# teeth

Rolling cylinders (idealized toothed gear set or friction drive)

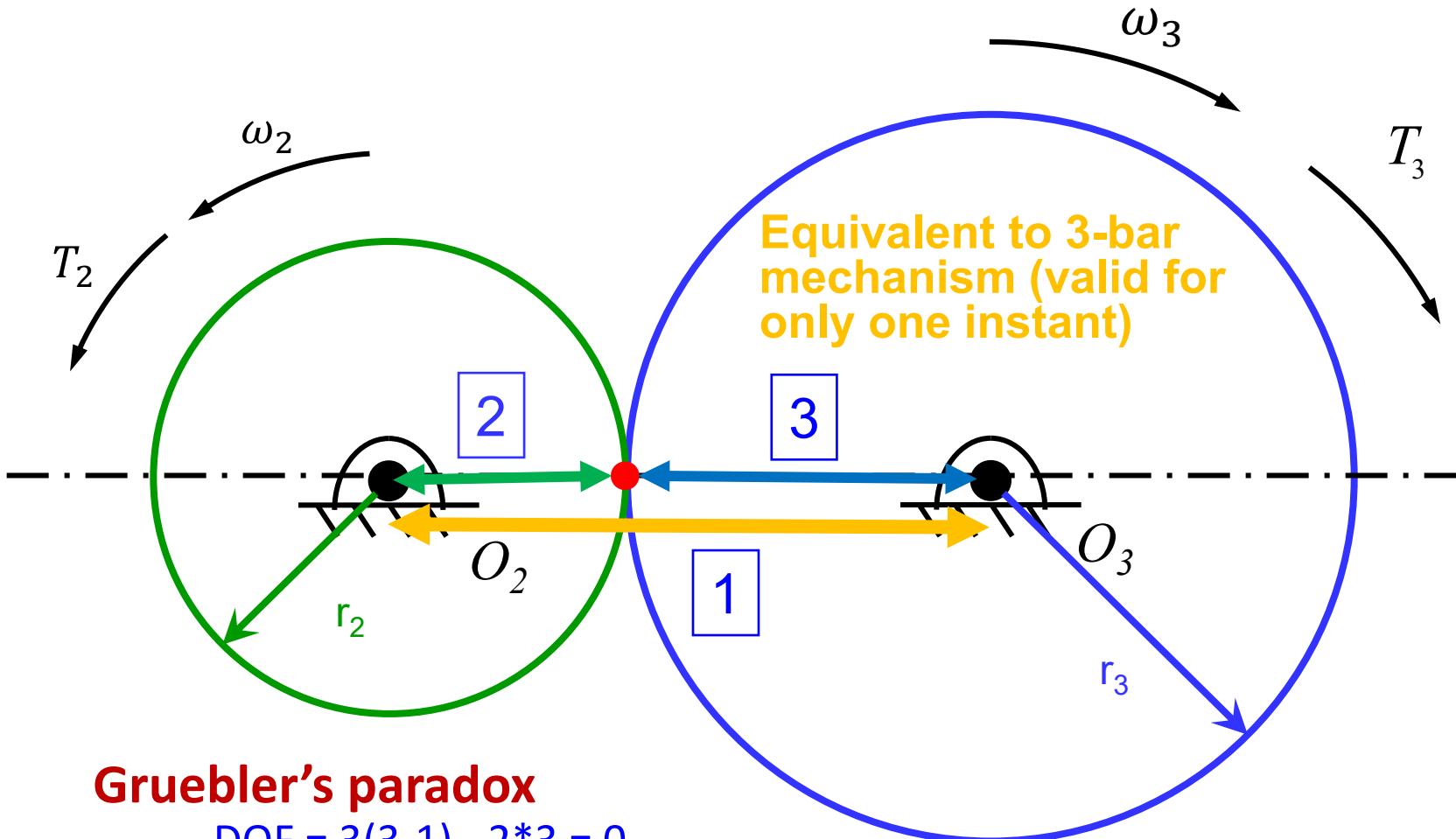
Key terms:

Pitch circle
Pitch diameter/radius
Pitch point



An external gear set

Recall Gruebler's paradox



An external gear set

$$DOF = 3(3-1) - 2*3 = 0$$

Gruebler's predicts no motion

Ground length = sum of two radii

Gruebler's equation does not account for link size or shape.

Moral : Watch out for higher symmetry (e.g., parallel links, summed length)

Velocity ratio: Modeling gears as rolling cylinders:

Surface *velocity* at the contact point, or *pitch point*, is the *same*:

Given:

$$r_{in} = 4 \text{ cm}$$

$$r_{out} = 6 \text{ cm}$$

$$\omega_{in} = 90 \text{ rpm}$$

$$\omega_{out} = ? = \frac{-r_{in}}{r_{out}} \omega_{in} = \frac{-4}{6} \cdot 90 \text{ rpm} = -60 \text{ rpm}$$

Angular velocity ratio (m_V):

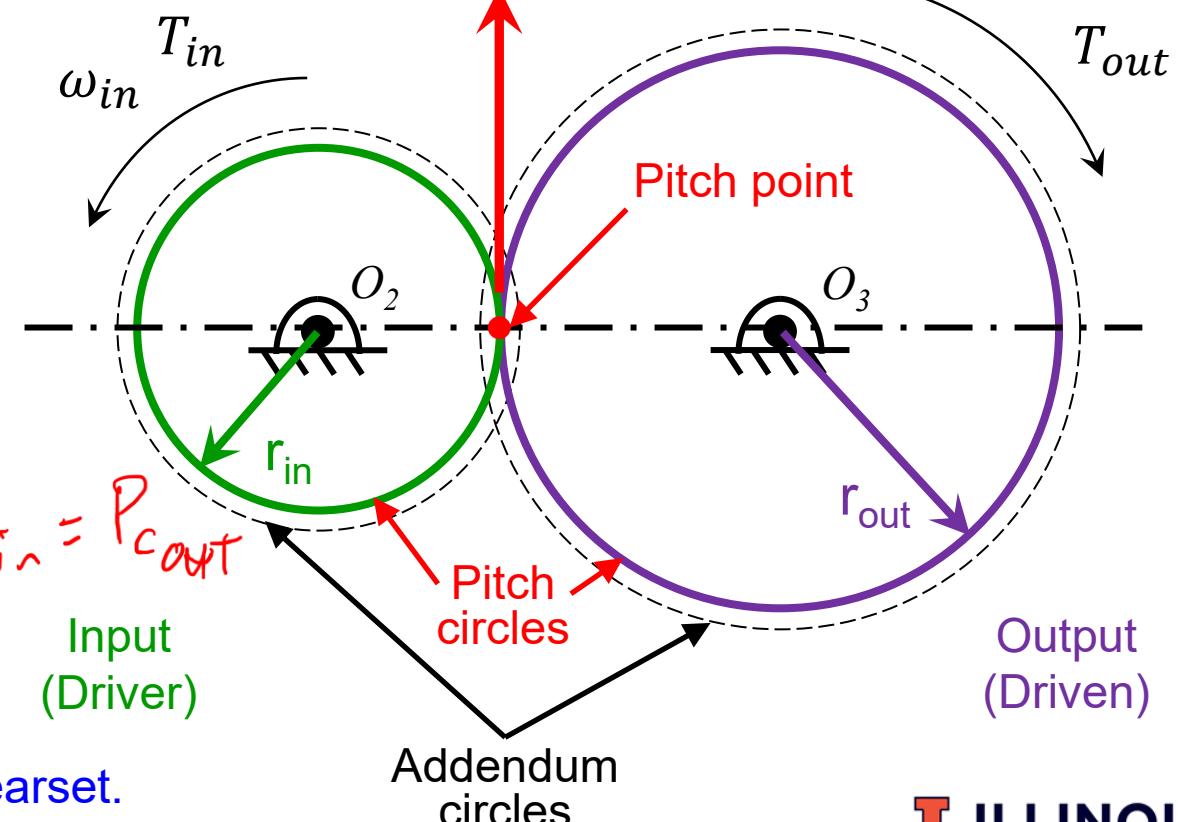
$$\therefore m_V = \pm \frac{\omega_{out}}{\omega_{in}} = \pm \frac{r_{in}}{r_{out}} = \pm \frac{d_{in}}{d_{out}} = \pm \frac{N_{in}}{N_{out}}$$

$$P_c = \frac{\pi d}{N}$$

$$P_{c_{in}} = P_{c_{out}}$$

Input
(Driver)

Addendum circles



Negative sign if **external gearset**, Positive sign if **internal gearset**.

Torque Ratio (m_T)

Tangential *force* at the contact point, or *pitch point*, is the *same*:

$$r_{in} = 4 \text{ cm}$$

$$r_{out} = 6 \text{ cm}$$

$$T_{in} = 1 \text{ Nm}$$

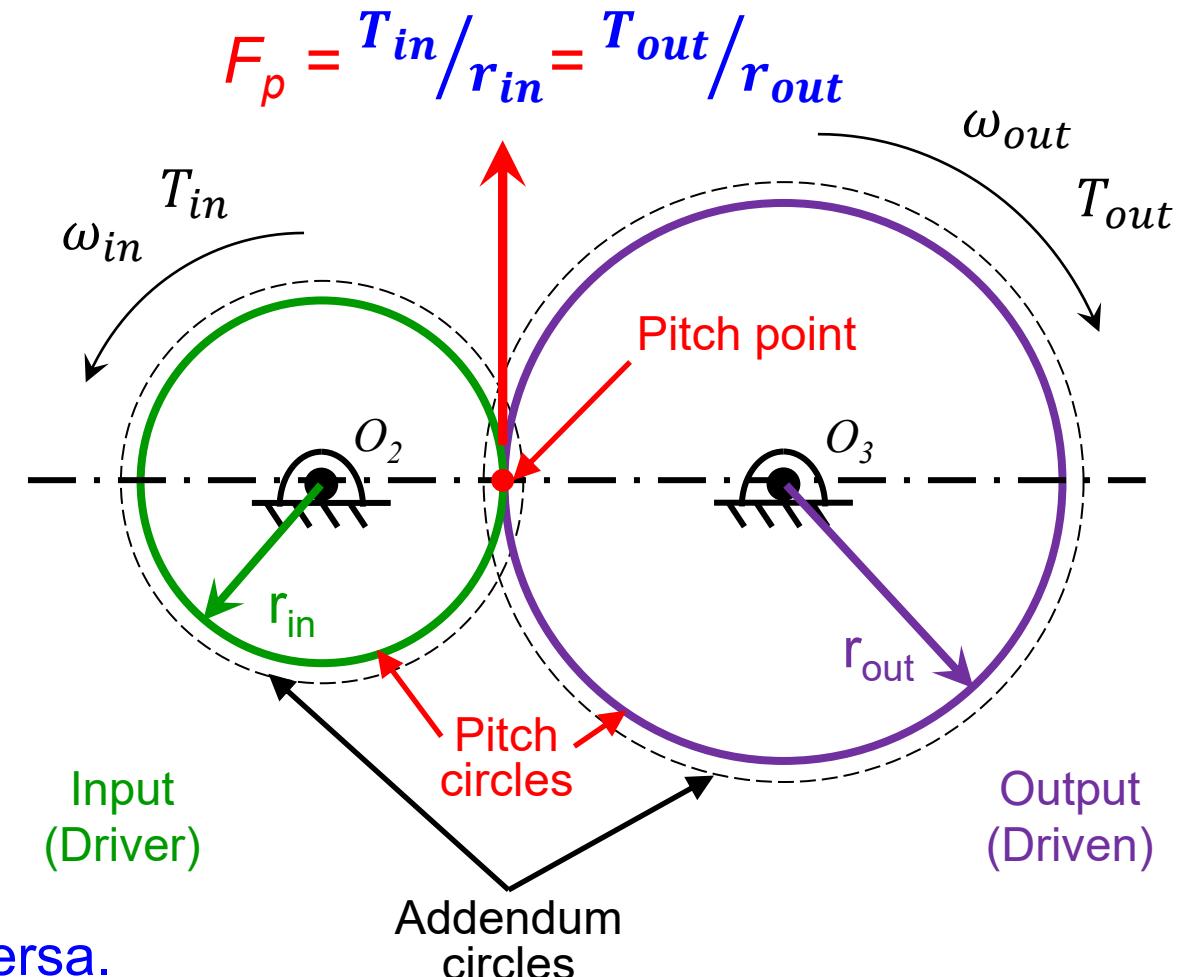
$$T_{out} = ? = \frac{r_{out}}{r_{in}} T_{in} = \frac{6}{4} \cdot 1 \text{ Nm} =$$

1.5 Nm

▪ Torque ratio (m_T):

$$\therefore m_T = \frac{T_{out}}{T_{in}} = \frac{\omega_{in}}{\omega_{out}} = \frac{1}{m_V}$$

Gearsets exchange torque for velocity or vice versa.



Power transmission

- Assuming a gearset is highly efficient, then $P_{out} = \sim P_{in}$

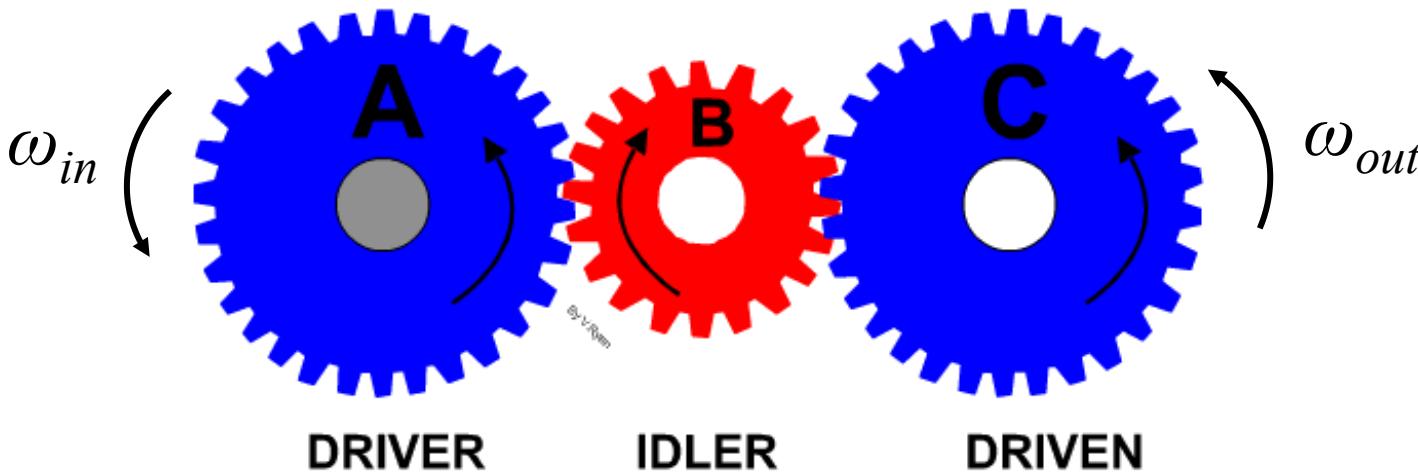
$$P = \vec{T} \cdot \vec{\omega} = \vec{T}_{in} \cdot \vec{\omega}_{in} = \sim \vec{T}_{out} \cdot \vec{\omega}_{out}$$

$$|m_V| \equiv \frac{\omega_{out}}{\omega_{in}} = \pm \frac{r_{in}}{r_{out}} = \pm \frac{d_{in}}{d_{out}} = \pm \frac{N_{in}}{N_{out}} = \frac{1}{|m_T|} \equiv \frac{T_{in}}{T_{out}}$$

Velocity Ratio
Gear Ratio

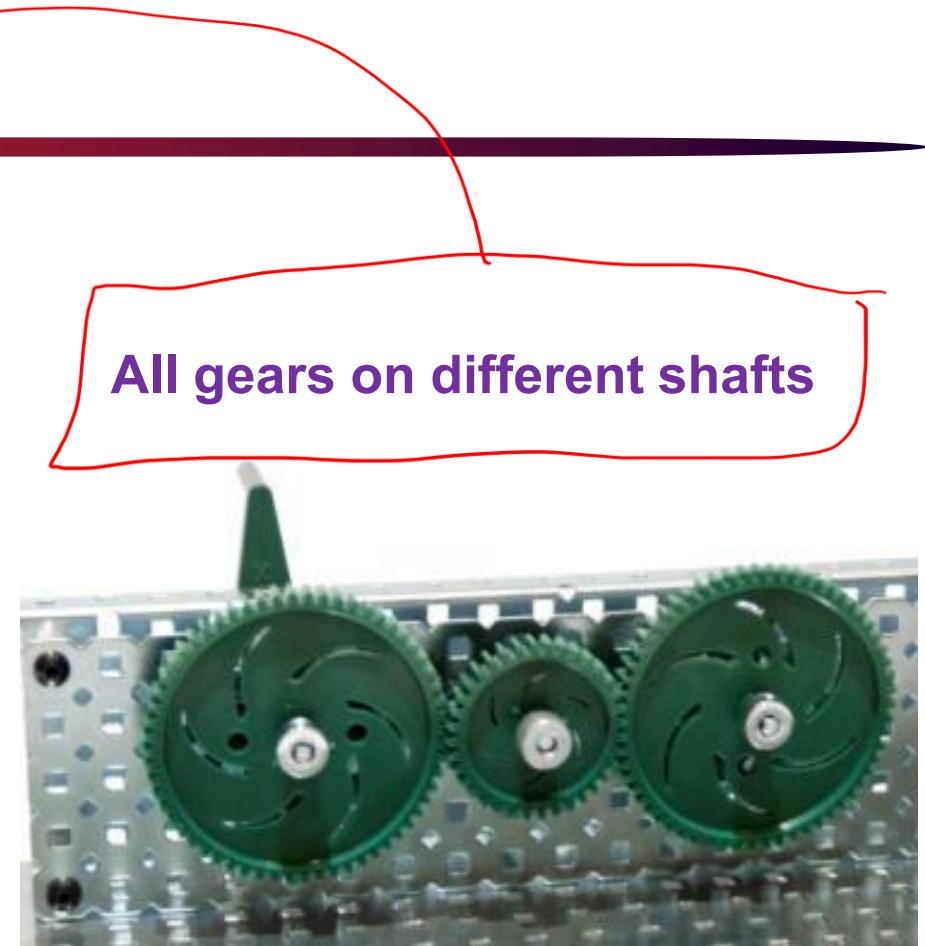
Torque Ratio

Simple gear train



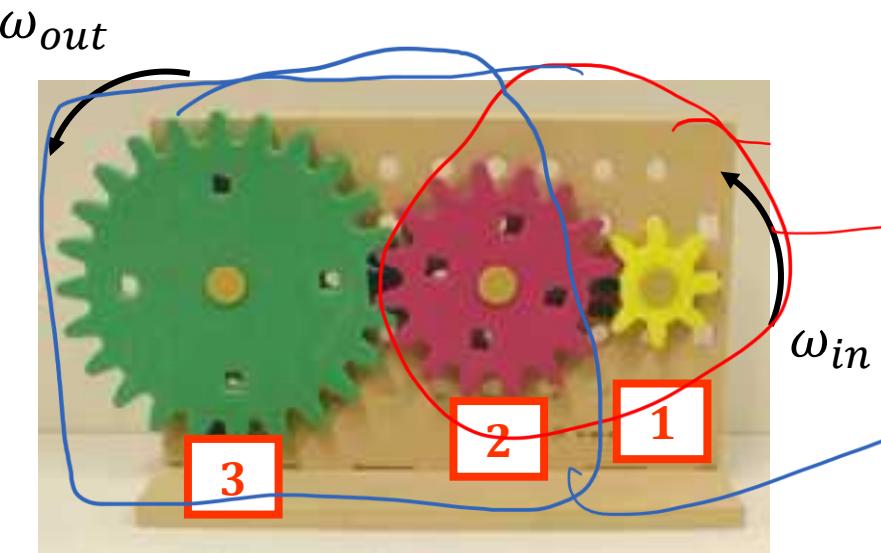
<https://technologystudent.com/gears1/gears2.htm>

- Middle gear is called an **idler** gear.
 - Has no contribution to gear ratio.
 - Only changes direction of input to output.
 - Odd # gears: input & output same direction
 - Even # gears: input & output opposite direction
- Typically, do not use more than one idler gear. If want to cover longer distances, then use chain or belt.



<https://quizlet.com/255781174/simple-gear-train-with-idler-diagram/>

Simple gear train



$$|m_V| \equiv \frac{\omega_{out}}{\omega_{in}} = \pm \frac{r_{in}}{r_{out}} = \pm \frac{d_{in}}{d_{out}} = \pm \frac{N_{in}}{N_{out}} = \frac{1}{|m_T|} \equiv \frac{T_{in}}{T_{out}}$$

$$m_V \equiv \frac{\omega_{out}}{\omega_{in}} = \left(-\frac{N_1}{N_2} \right) \left(-\frac{N_2}{N_3} \right) = \left(\frac{N_1}{N_3} \right) = \left(\frac{8}{24} \right) = \frac{1}{3}$$

$$N_3 = 24 \quad N_2 = 16 \quad N_1 = 8$$

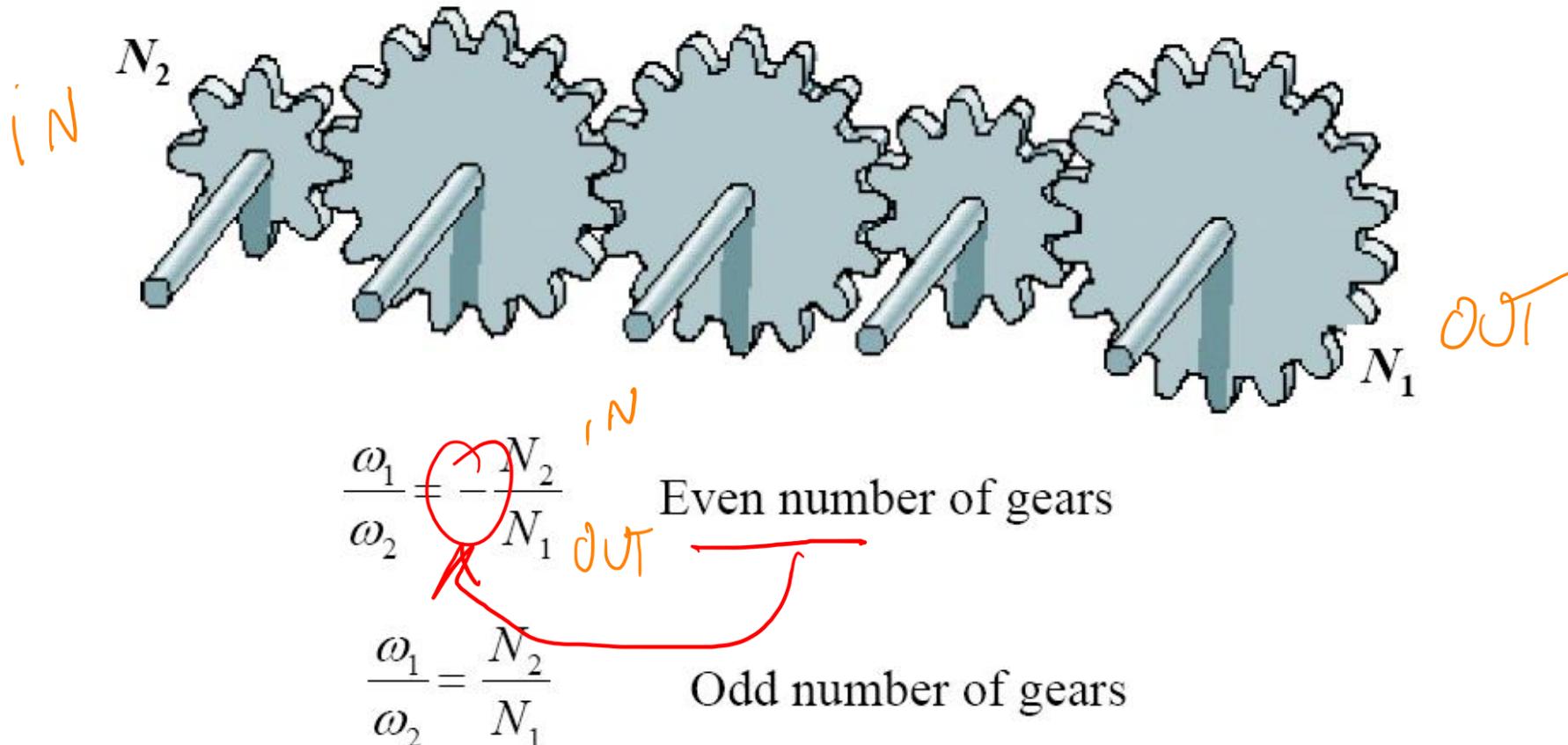
$$\omega_{in} = 90 \text{ rpm} \quad \omega_{out} = 30 \text{ rpm}$$

$$T_{in} = 1 \text{ Nm} \quad T_{out} = 3 \text{ Nm}$$

Negative because external gear trains

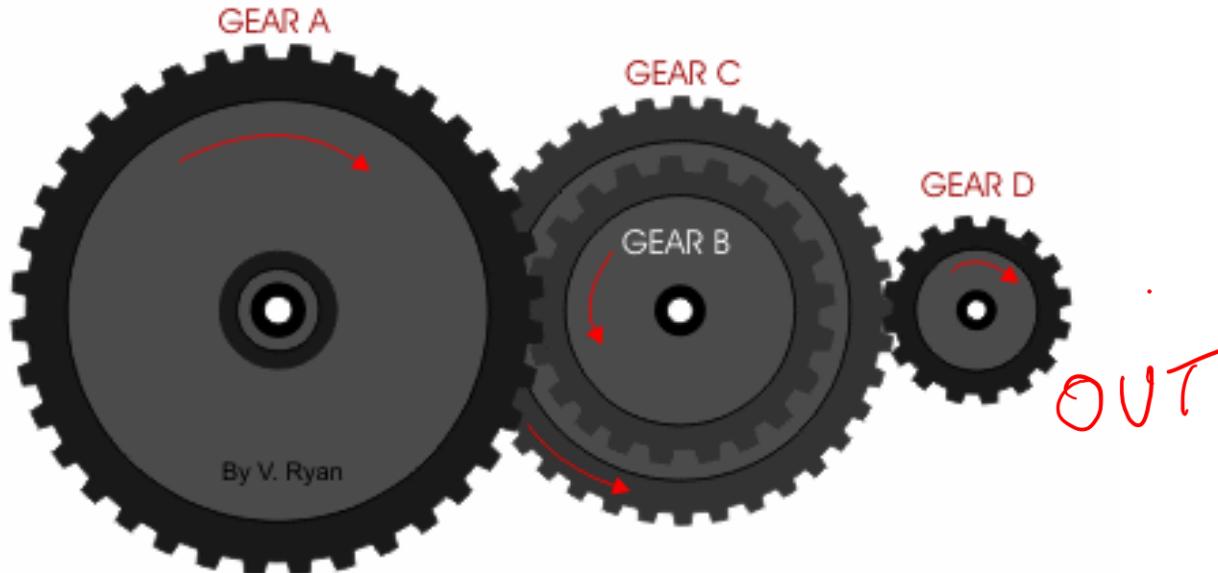
$$m_T \equiv \frac{T_{out}}{T_{in}} = \frac{1}{m_V} = 3$$

Longer simple gear trains



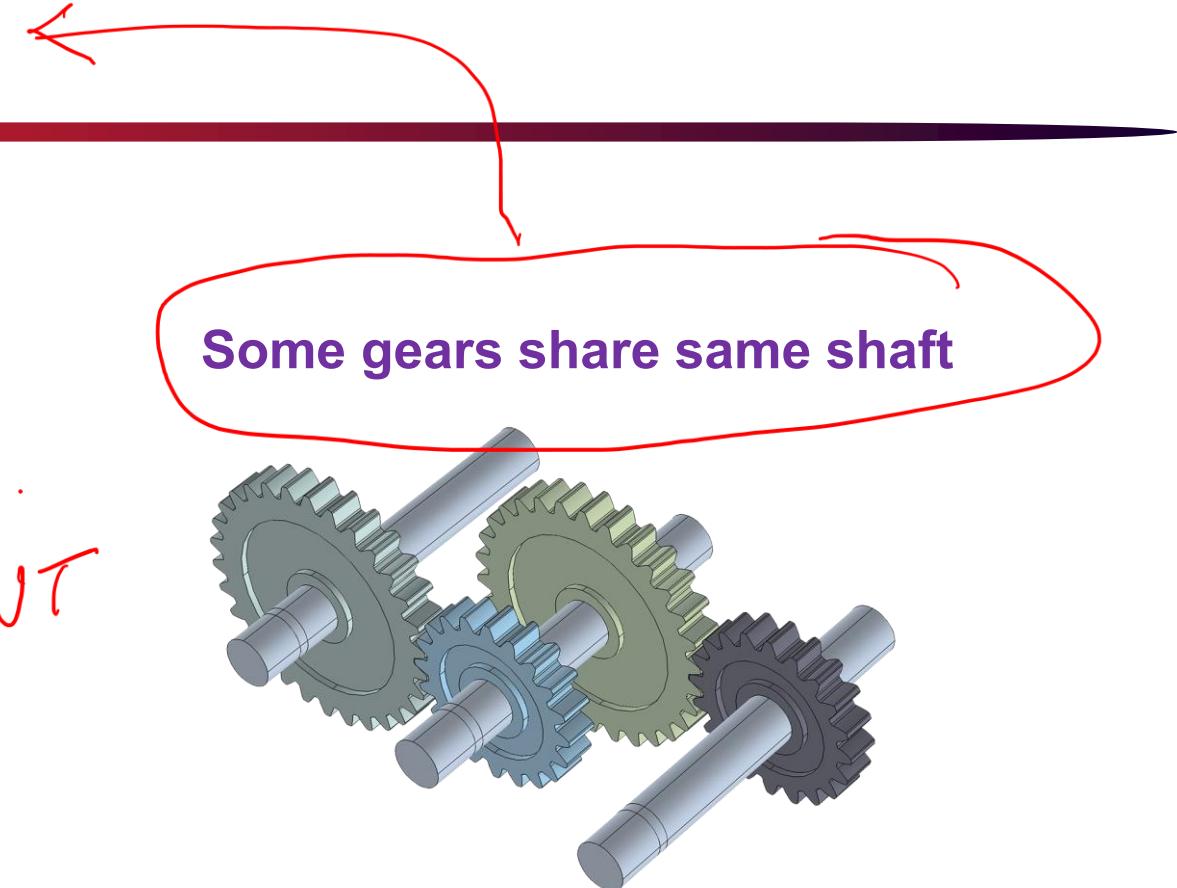
Compound gear trains

IN



<http://www.technologystudent.com/gears1/gears8.htm>

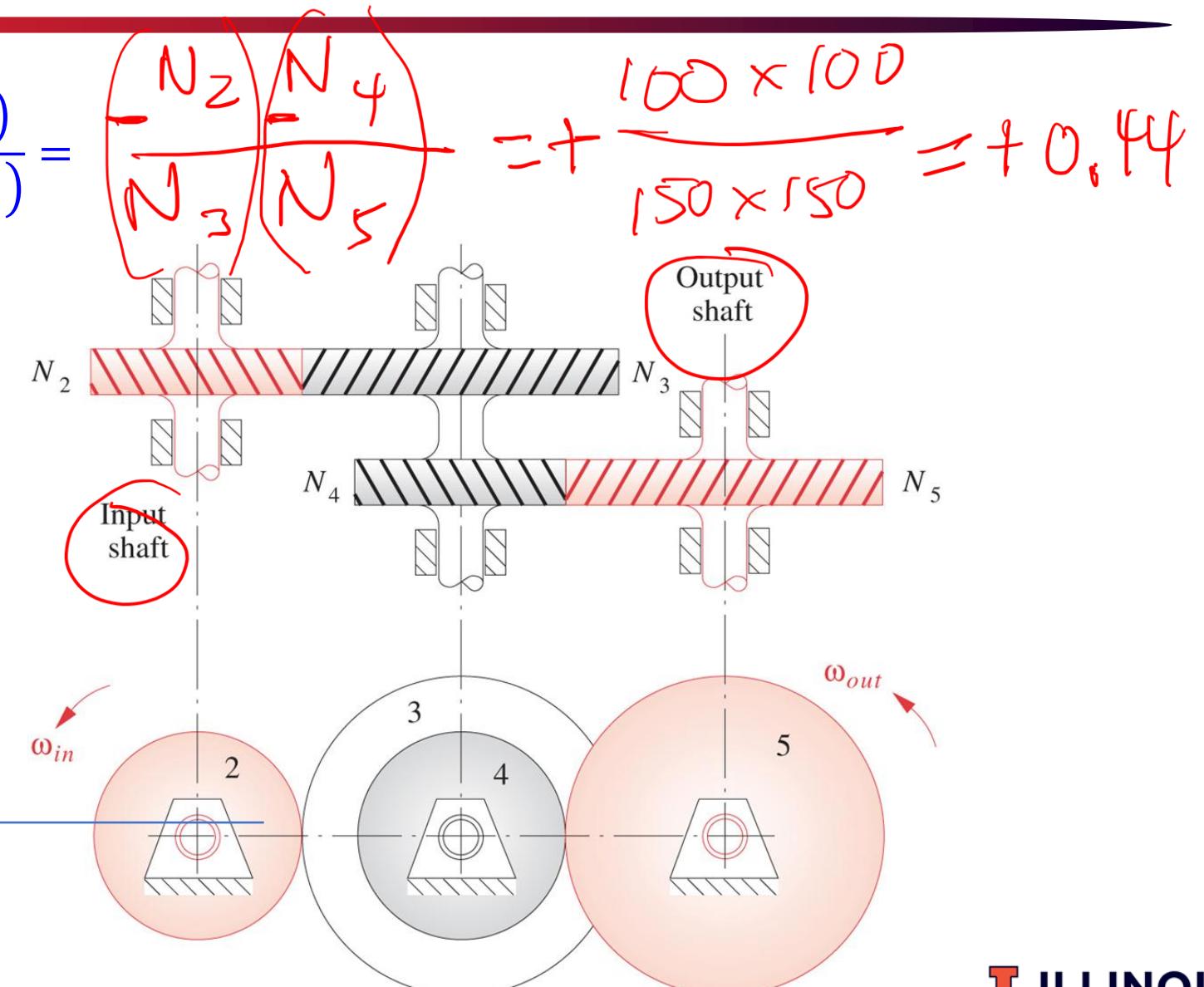
A & C are **drivers** (in)
B & D are **driven** (out)



<https://images.app.goo.gl/PwfqXyj7cR33pdzF8>

Compound gear trains

$$m_V = \pm \frac{\prod(N_{in \text{ or } "driver \text{ gears"}})}{\prod(N_{out \text{ or } "driven \text{ gears"}})}$$

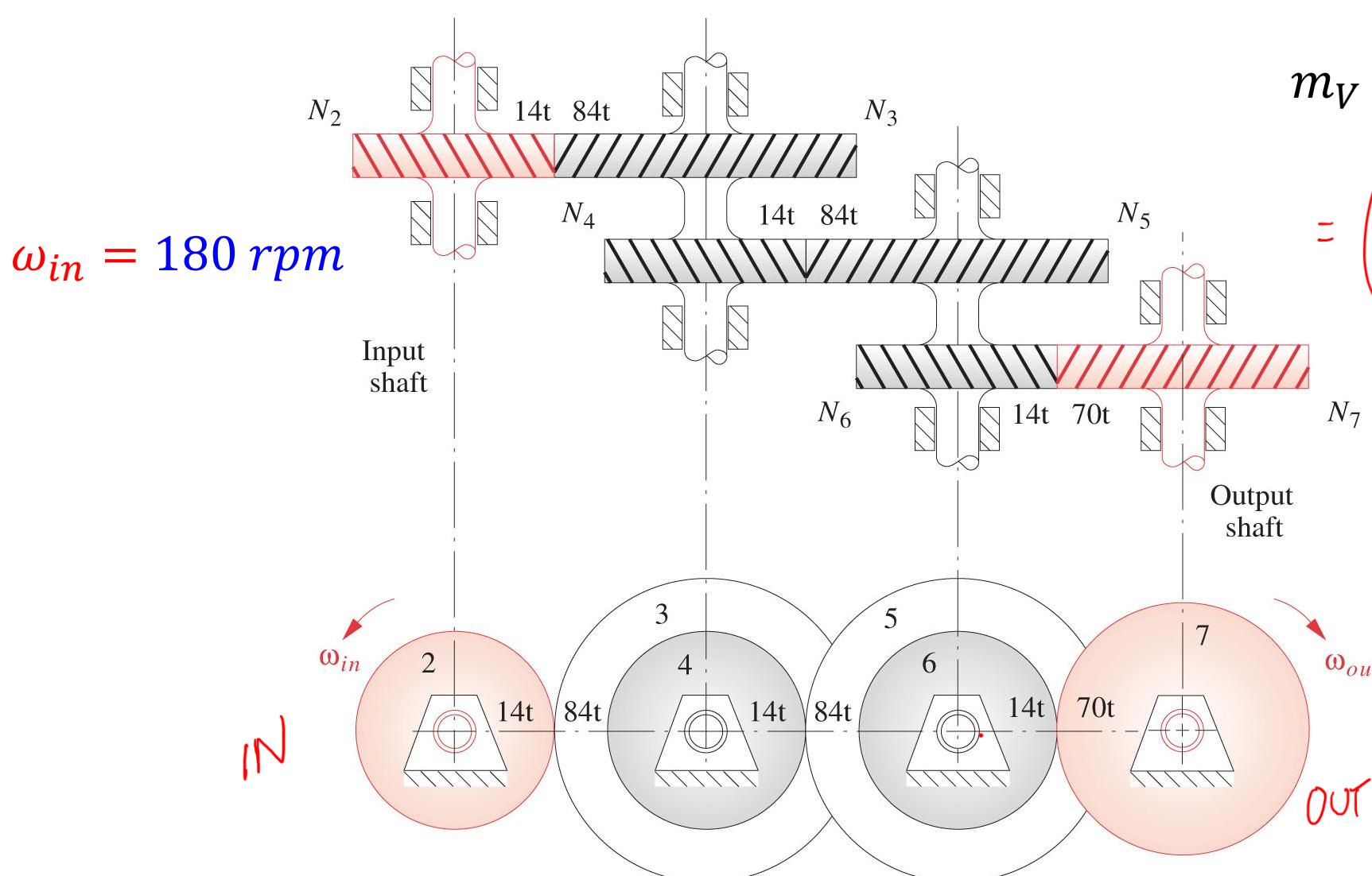


Some gears share
the same shaft:

2 & 4 are drivers (*in*)
3 & 5 are driven (*out*)

$$N_2 = N_4 = 100$$
$$N_3 = N_5 = 150$$

What is the output rpm?



$$m_V = \pm \frac{\prod (N_{\text{driver gears}})}{\prod (N_{\text{driven gears}})}$$

$$= \left(-\frac{N_2}{N_3} \right) \left(-\frac{N_4}{N_5} \right) \left(-\frac{N_6}{N_7} \right)$$

$$= -0.0056$$

$$m_V \equiv \frac{\omega_{out}}{\omega_{in}}$$

$$\omega_{out} = -0.0056 * 180$$

$$= -1 \text{ rpm}$$