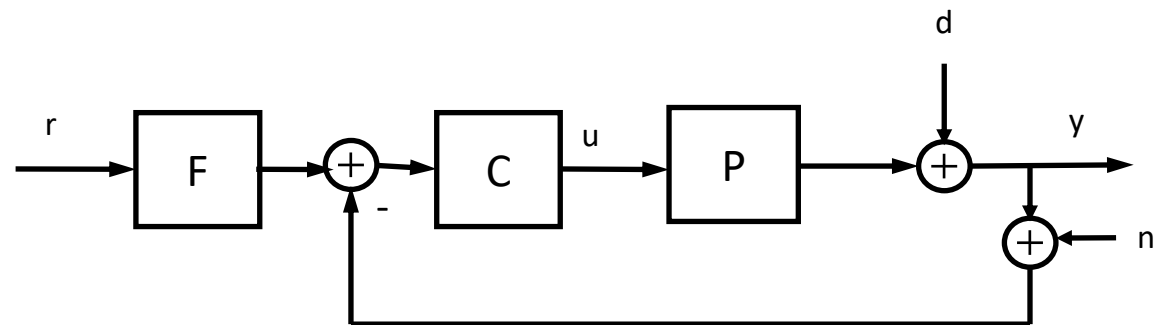


# Summary

## Themes

- Feedback
- Stability
- Robustness
- Performance
  - Frequency response
  - Loop shaping



# Summary of Feedback

## Benefits

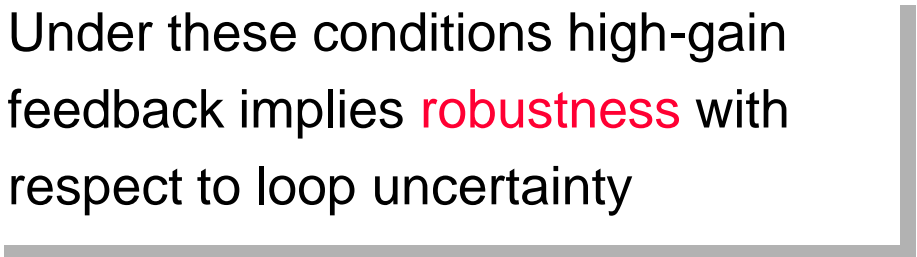
- disturbance rejection
- robustness
- linearity improvement
- bandwidth improvement

## Closed-loop Stability Required

Good tracking and disturbance attenuation are retained as long as

- closed-loop system remains stable
- the gain remains high

Under these conditions high-gain feedback implies **robustness** with respect to loop uncertainty



# Pitfalls of High Gain Feedback

## Potential problems

- naively making the gain large can easily result in an **unstable** feedback system
- even if feedback system is stable, overly large plant inputs may arise that exceed the plant capacity
- measurement noise causes loss of performance

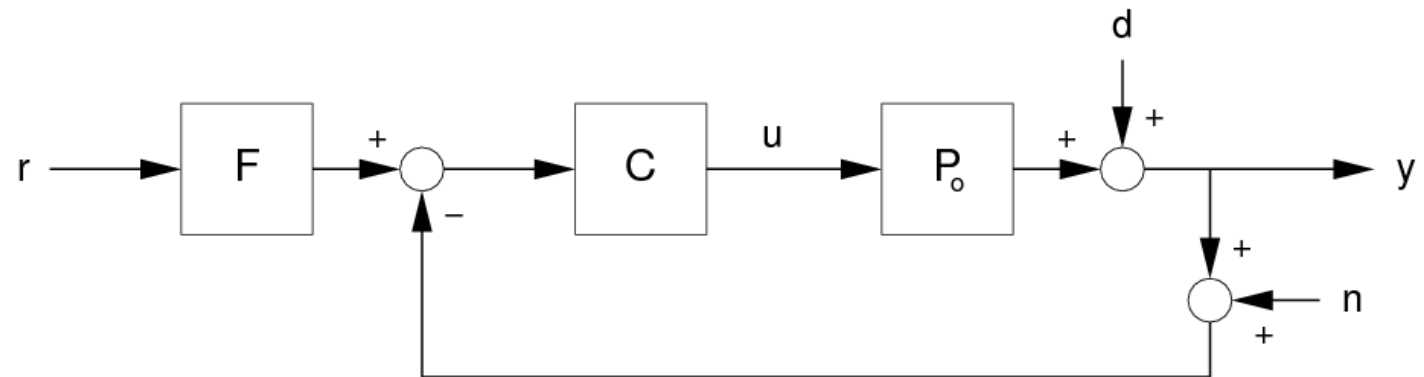
## Design Issues (recap)

### Targets

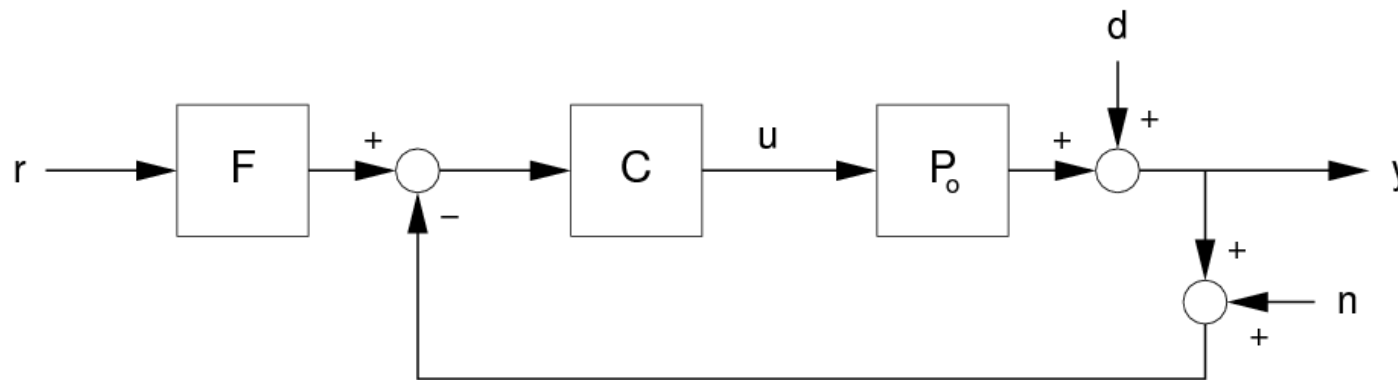
- Closed-loop stability
- Disturbance attenuation
- Good command response
- Robustness
  - stability
  - performance

### Limitations

- Plant capacity
- Measurement noise



## System functions: $L$ and $S$



**Loop gain  $L$**

$$L = PC$$

▪ **Sensitivity function  $S$**

$$y = \frac{1}{1+L} d$$

$S$

## Disturbance Attenuation and Bandwidth

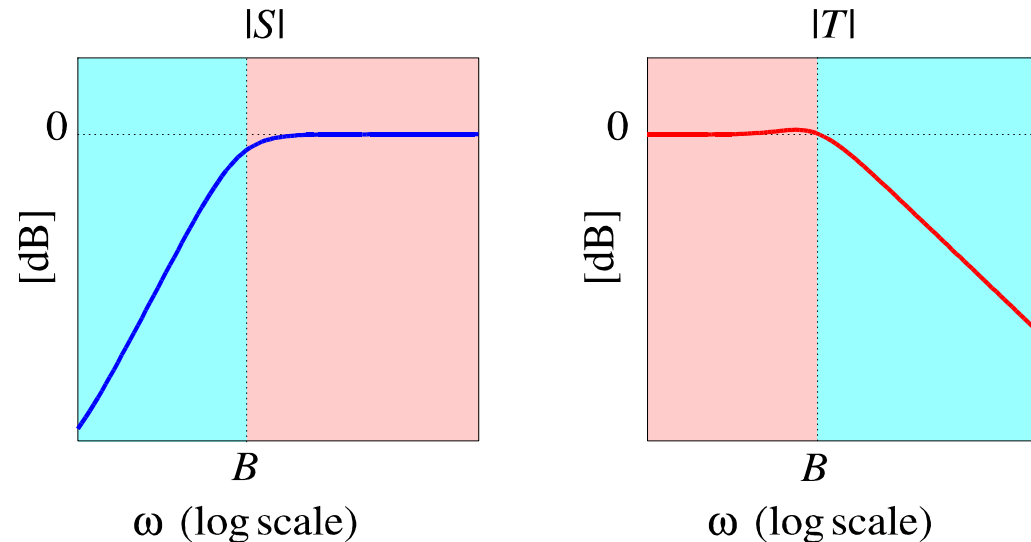
### Disturbance Attenuation

- The smaller  $|S(j\omega)|$  is the more disturbances are **attenuated** at frequency  $\omega$
- $|S|$  is small if the magnitude of the loop gain  $L$  is large
- $L$  needs to be made large for frequencies where disturbance attenuation is needed
- However, this is limited by **plant capacity**

### Bandwidth

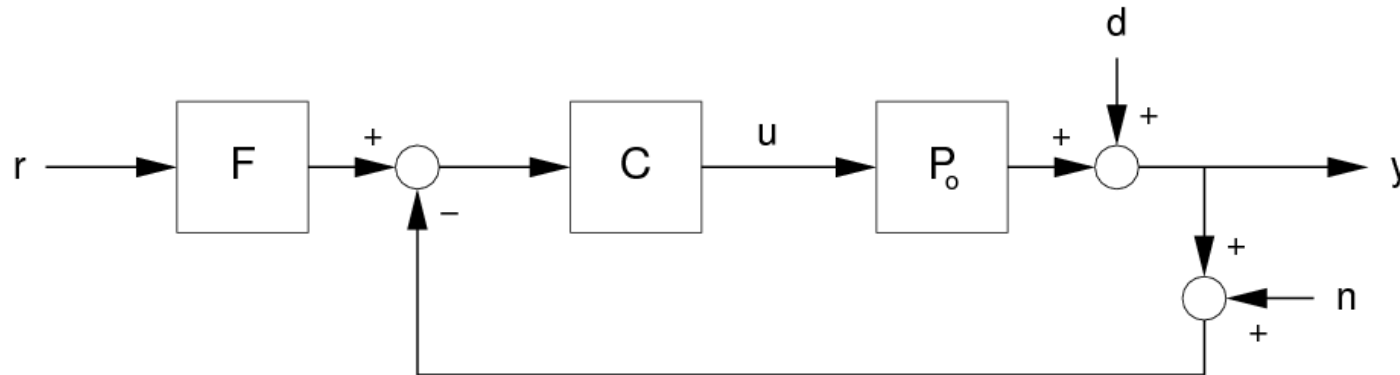
- $L$  can only be made large over a limited frequency band
- The size of this band is called the **bandwidth**  $B$

## Bandwidth and Crossover Region (S)



- Typical shape of magnitude of the sensitivity function:
  - low at frequencies up to the bandwidth
  - near 1 at frequencies above the bandwidth
- The frequency range around  $B$  is the **crossover region**
  - “peaking” of  $S$  should be avoided
  - otherwise disturbances are amplified

System functions:  $TF$  and  $T$



▪ **Closed-loop transfer function  $TF$**

$$y = \underbrace{\frac{L}{1+L}}_{TF} F r$$

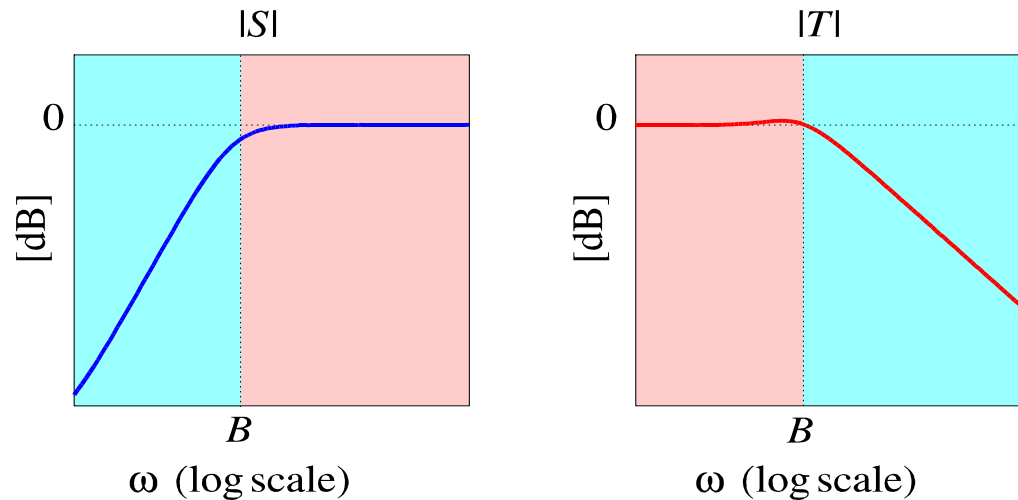
**Complementary sensitivity function  $T$**

$$TF = \frac{L}{1+L} F$$

$$T$$

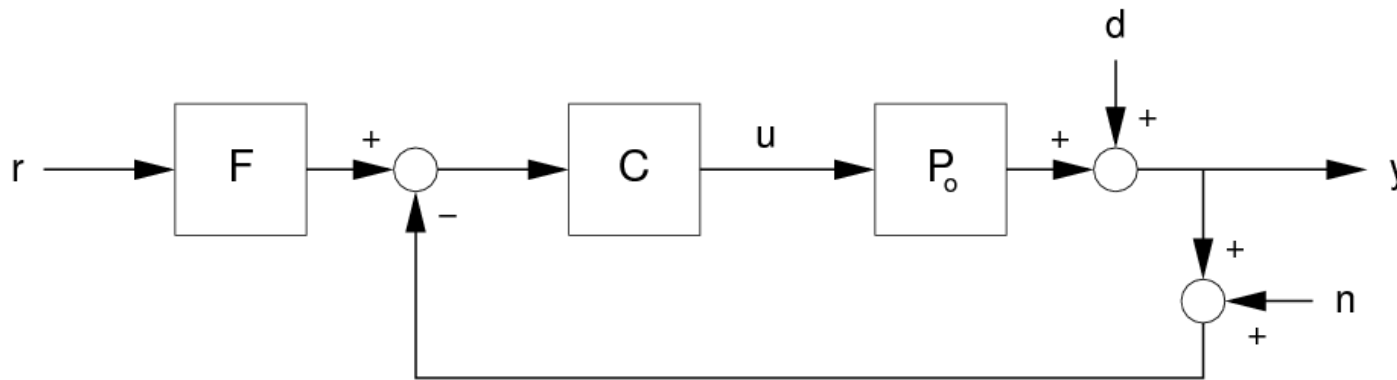


## Command Response (T,F)



- Recall that  $T = 1 - S$
- $T$  determines the **command response** - it is close to 1 up to  $B$
- When  $F = 1$ , closed-loop transfer function  $TF$  is low pass with the same bandwidth as the band for disturbance attenuation
- If a different command response is required,  $F$  can **compensate** for this

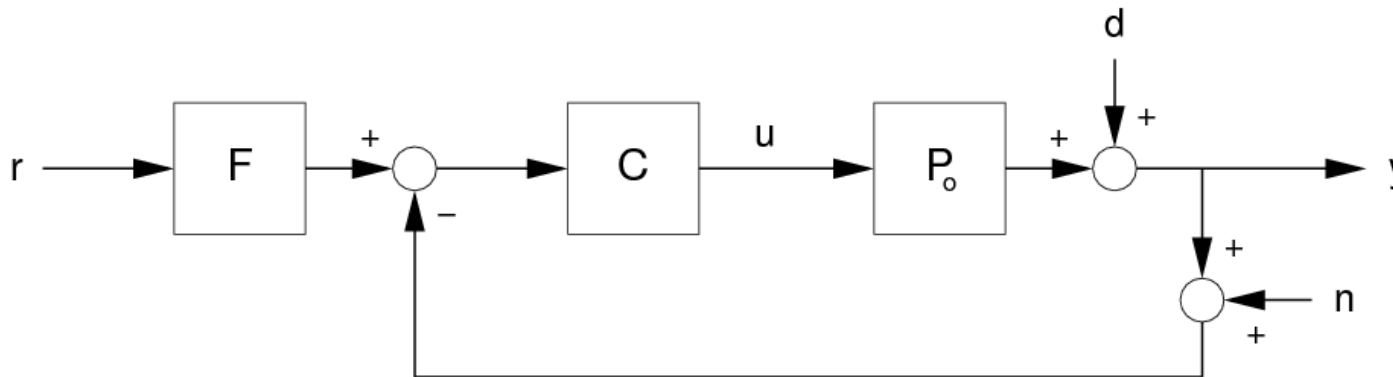
## Measurement noise (T)



- T determines measurement noise sensitivity
- high frequencies: T should decrease as quickly as possible
- low frequencies: T is close to 1 - this emphasises the need for good low-noise sensors

$$y = \underbrace{\frac{1}{1+PC}}_S d + \underbrace{\frac{PC}{1+PC}}_T Fr - \underbrace{\frac{PC}{1+PC}}_T n$$

System functions:  $S_u$



- **Control sensitivity function  $S_u$**

$$u = \underbrace{\frac{C}{1+CP}}_{S_u} (Fr - n - d)$$

## Plant Capacity ( $S_u$ )

Note that  $T = S_u P$

Thus, requirements on  $S_u$  can be translated into requirements on  $T$

- To **prevent overly large inputs**  $S_u(T)$  should not be too large
- At low frequencies the loop gain should be high for low sensitivity, and the magnitude of  $T$  is close to 1
- This can lead to **plant capacity** being exceeded
- At high frequencies  $S_u$  should decrease as fast as possible, otherwise measurement noise affects the input - this is consistent with the robustness requirement that  $T$  decrease fast

## Plant Capacity ( $S_u$ ) - r.h.p. zeros

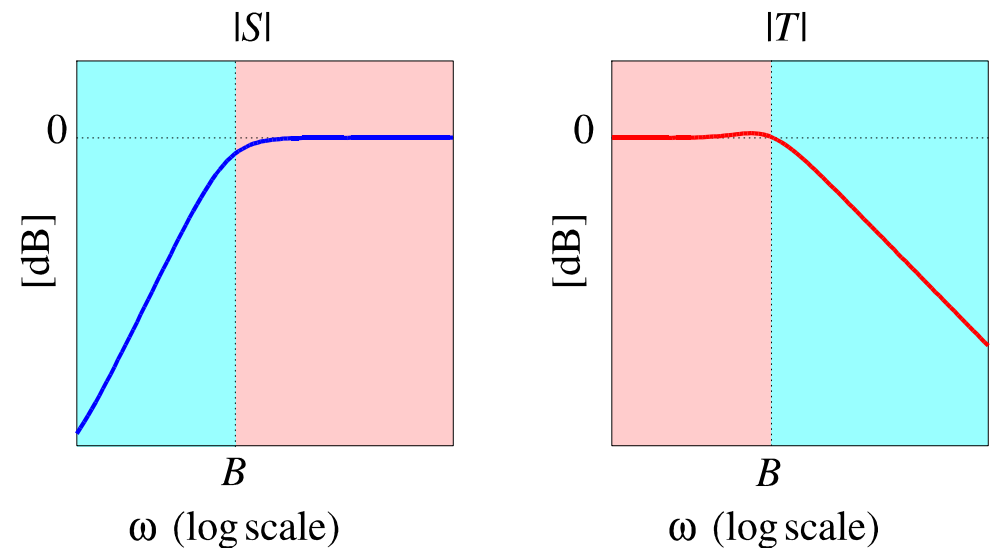
When  $L = CP \gg 1$  then  $S_u \approx \frac{1}{P}$

If the plant  $P$  has zeros in the right half plane,  $1/P$  is unstable

- Unstable open loop plant zeros limit the closed-loop bandwidth
- $S_u$  may only be made equal to  $1/P$  up to the frequency which equals the magnitude of the r.h.p. zero with the smallest magnitude

# Stability Robustness

- **Robustness**
  - For loop gain perturbations  $T$  needs to be small
  - For inverse loop gain perturbations  $S$  needs to be small
- **Performance**
  - At high frequencies  $T$  needs to be small
  - At low frequencies  $S$  needs to be small
- **Perturbations**
  - High frequency uncertainty (parasitics) causes significant loop gain perturbations
  - low frequency uncertainty (load changes etc.) causes significant inverse loop gain perturbations
- **Crossover**
  - Neither  $S$  nor  $T$  can be small, they must therefore be **prevented from peaking**.  
Good stability margins help to ensure this



	$S$	$T$
low frequencies	small	$\cong 1$
high frequencies	$\cong 1$	small

## Performance Robustness

- Performance is determined by  $S$ ,  $T$ ,  $S_u$  and  $TF$
- Since  $S_u$  and  $TF$  depend on  $S$  and  $T$ ,  
we need only consider the effect of perturbations on  $S$  and  $T$
- For robust  $S$  we need  $T_o$  small
- For robust  $T$  we need  $S_o$  small
- Normally,  $S$  is small at low frequencies, making  $T$  robust at low frequencies - this is the region where  $T$ 's values are significant
- Conversely,  $T$  is normally small at high frequencies, making  $S$  robust at high frequencies - the region where  $S$  is significant

Denote nominal quantities by  $S_o$  etc.

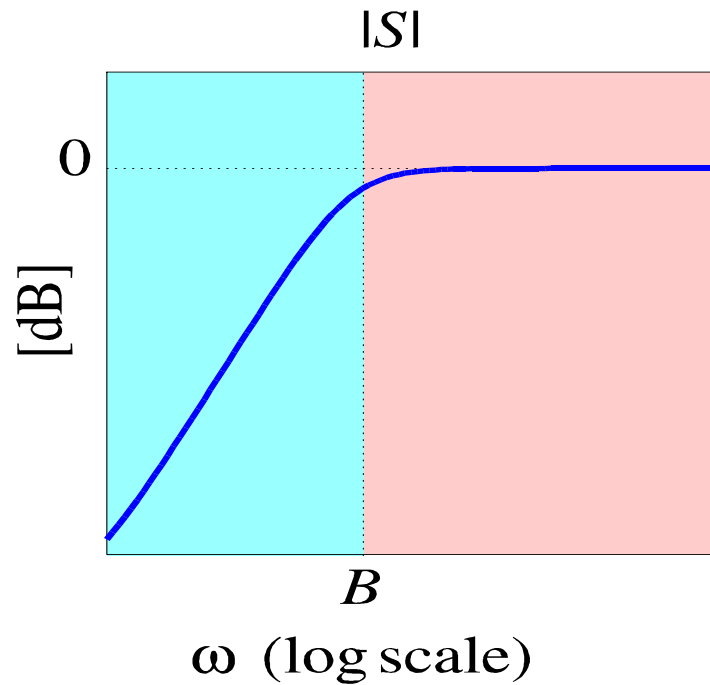
Relative changes in  $1/S$  and  $1/T$

$$\frac{\frac{1}{S} - \frac{1}{S_o}}{\frac{1}{S_o}} = \frac{S_o - S}{S} = T_o \frac{L - L_o}{L_o} = T_o \frac{\frac{1}{L_o} - \frac{1}{L}}{\frac{1}{L}}$$

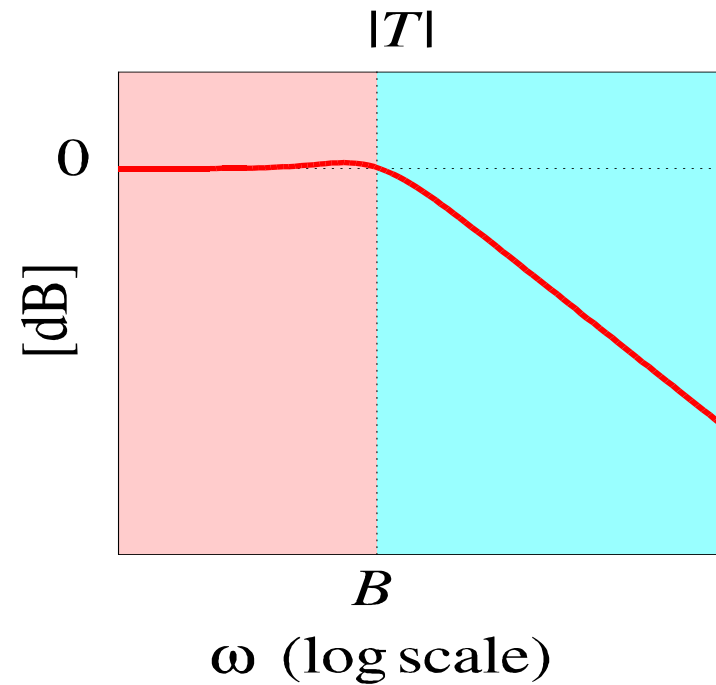
$$\frac{\frac{1}{T} - \frac{1}{T_o}}{\frac{1}{T_o}} = \frac{T - T_o}{T} = S_o \frac{L - L_o}{L} = S_o \frac{\frac{1}{L} - \frac{1}{L_o}}{\frac{1}{L_o}}$$

Vector margin related to peak S

Complementary vector margin related to peak T



$$s_m = \frac{1}{\max_{\omega} |S_o(j\omega)|}$$



$$r_m = \frac{1}{\max_{\omega} |T_o(j\omega)|}$$



## Review of Design Requirements

Sensitivity  $S$  small at low frequency to achieve

- disturbance attenuation
- good command response
- robustness at low frequencies

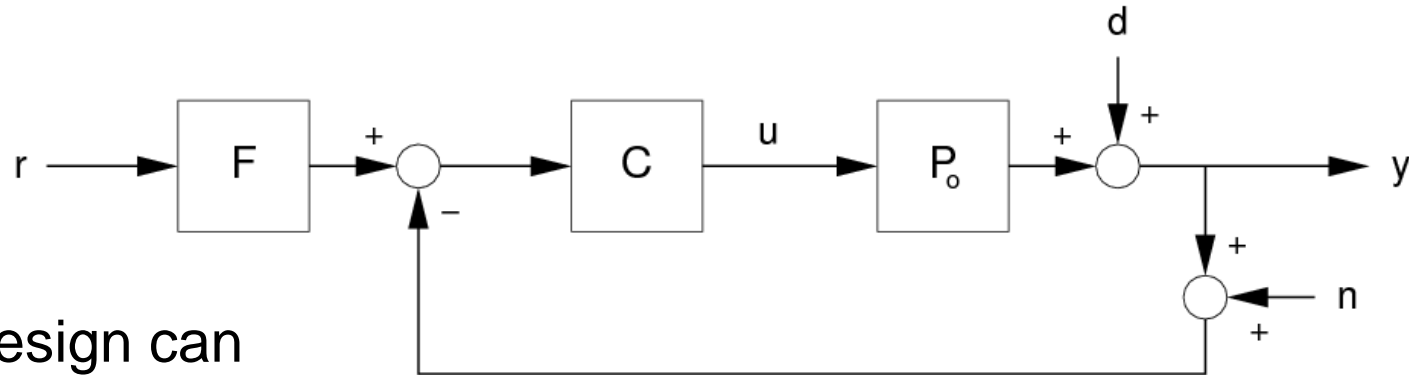
Complementary sensitivity  $T$  small at high frequencies to prevent

- exceeding plant capacity
- adverse effects of measurement noise
- loss of robustness at high frequencies

In the Crossover Region peaking of both  $S$  and  $T$  should be avoided to prevent

- overly large disturbance sensitivity
- excessive influence of measurement noise
- loss of robustness

## Loop Gain L



- Feedback system design can be seen as a process of **loop shaping**
- Performance and robustness requirements result in specifications on  $|S|$  in the low frequency region and on  $|T|$  in the high frequency region
- This results in bounds on the loop gain L

$$S = \frac{1}{1+L}, \quad T = \frac{L}{1+L}, \quad L = CP$$

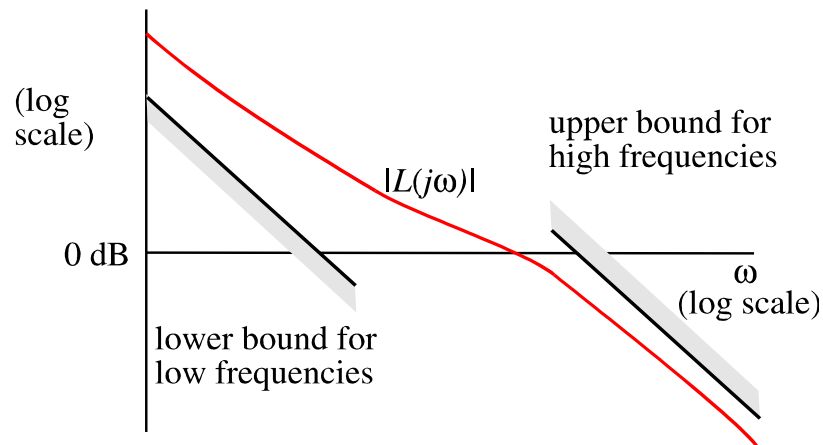
Low frequencies :

$$\text{require } S \ll 1, T \approx 1 \Leftrightarrow |L(j\omega)| \gg 1$$

High frequencies :

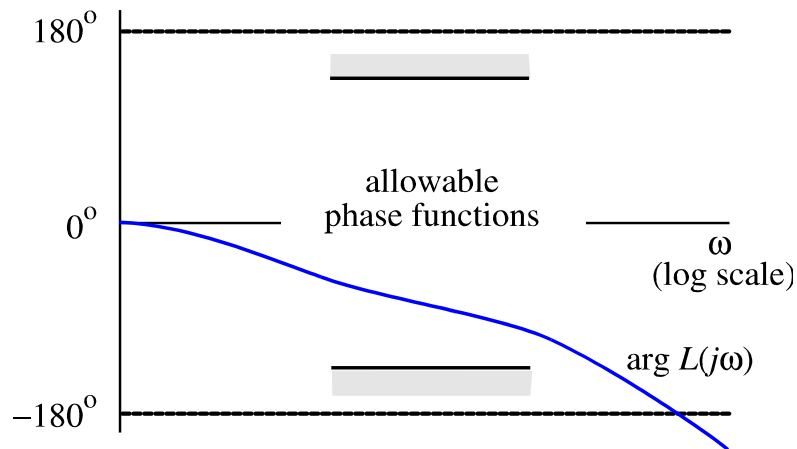
$$\text{require } T \ll 1, S \approx 1 \Leftrightarrow |L(j\omega)| \ll 1$$

# Loop shaping



Low frequencies: large loop gain

High frequencies: small loop gain



- In the crossover region the phase is constrained because of stability

## Crossover Region

- The more closely the Nyquist plot of  $L$  approaches  $-1$  the more **S peaks**
- If the plot of  $L$  approaches  $-1$  so does the plot of  $1/L$ . Hence the more closely the plot of  $L$  approaches  $-1$  the more **T peaks**
- Thus to avoid peaking we need **good stability margins**
- But gain and phase are not independent

$$S = \frac{1}{1 + L},$$
$$T = \frac{L}{1 + L} = \frac{1}{1 + \frac{1}{L}}$$