

$$1. \quad G_c(s) = K \quad G_p(s) = \frac{1}{s(s+8)}$$

$$L(s) = G_c(s) \cdot G_p(s) = K \frac{1}{s(s+8)}$$

$$N(s) = 1$$

$$n = 2$$

 $\Rightarrow$ 

$$m = 0$$

$$D(s) = s(s+8)$$

$$r = n - m = 2 \Rightarrow 2 \text{ asymptotes}$$

No zeros.

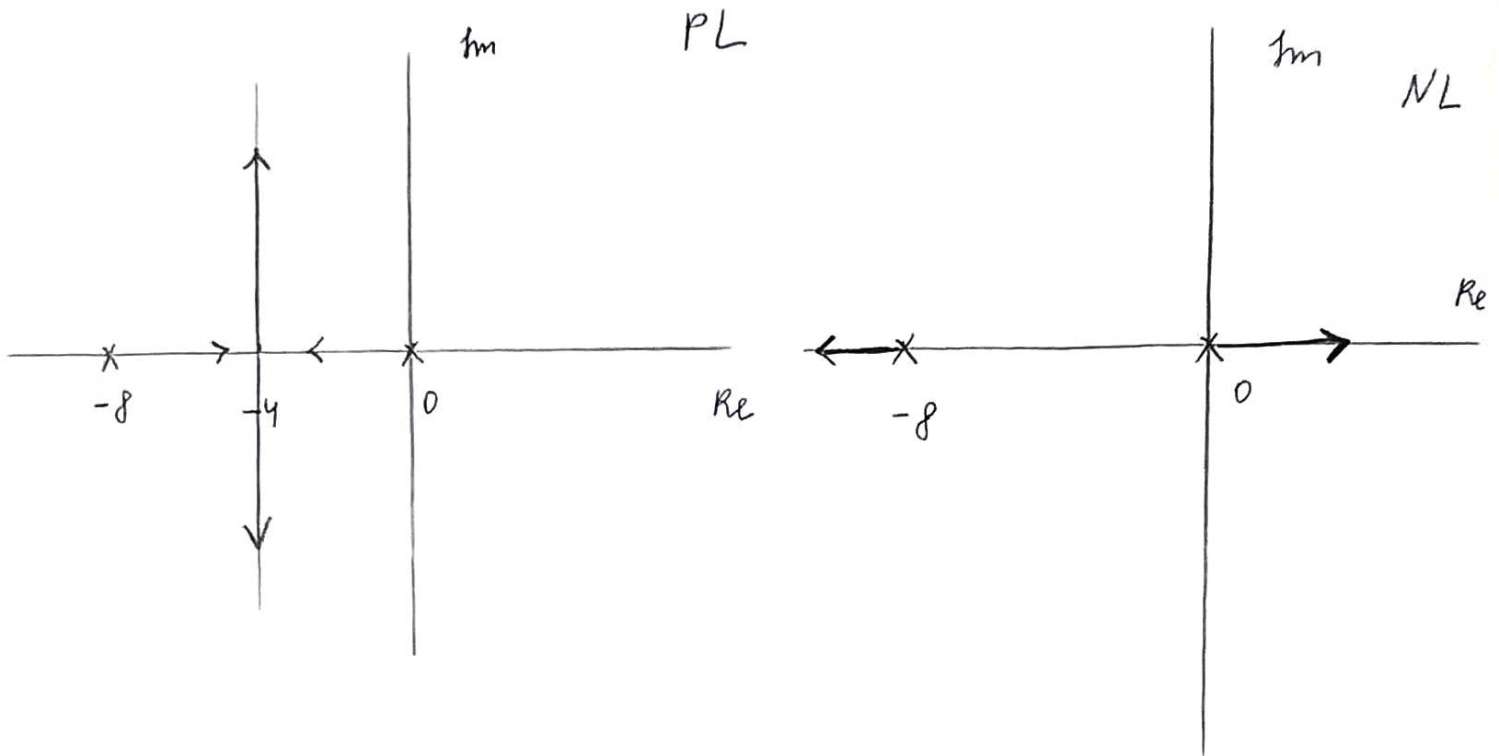
Poles at 0 and -8.

$$\sigma_a = \frac{0 - (0+8)}{2} = -4$$

$$\angle \psi_a = \begin{cases} (2h+1)\pi & h=0,1 & \text{PL} \\ 2h\pi & h=0,1 & \text{NL} \end{cases}$$

$$\Rightarrow \psi_a = \begin{cases} \frac{\pi}{2}, \frac{3\pi}{2} & \text{PL} \\ 0, \pi & \text{NL} \end{cases}$$

Real axis:  $(-\infty, -8)$  2 poles  $\in \text{NL}$  $(-8, 0)$  1 pole  $\in \text{PL}$  $(0, +\infty)$  0 pole  $\in \text{NL}$



Design requirements:  $M_p \leq 20\%$

$$T_r \leq 0.333 \text{ s}$$

$$T_{s, 2\%} \leq 2 \text{ s}$$

$$T_{s, 2\%} = \frac{4}{\zeta \omega_n} \leq 2 \Rightarrow \zeta \omega_n \geq 2$$

$$\bar{\zeta} = \frac{-\ln(M_p/100)}{\sqrt{\pi^2 + \ln^2(M_p/100)}} = \frac{-\ln(20/100)}{\sqrt{\pi^2 + \ln^2(20/100)}} = 0.456$$

$$\Rightarrow \zeta \geq 0.456$$

$$T_r = \frac{1.8}{\omega_n} \leq 0.333 \Rightarrow \omega_n \geq 5.4054$$

all requirements can be satisfied with a proportional controller.

let  $\zeta \omega_n = 4$ , therefore, the real part of a test point is equal to  $-4$ .

We need to find the imaginary part of the test point, knowing that  $\omega_n = \sqrt{[\operatorname{Re}(s^*)]^2 + [\operatorname{Im}(s^*)]^2}$

( $s^*$  - a test point)

$\omega_n \geq 5.4054$       Choose  $\omega_n = 7$

$$7 = \sqrt{(-4)^2 + [\operatorname{Im}(s^*)]^2}$$

$$\Rightarrow \operatorname{Im}(s^*) = \sqrt{33} \approx 5.745$$

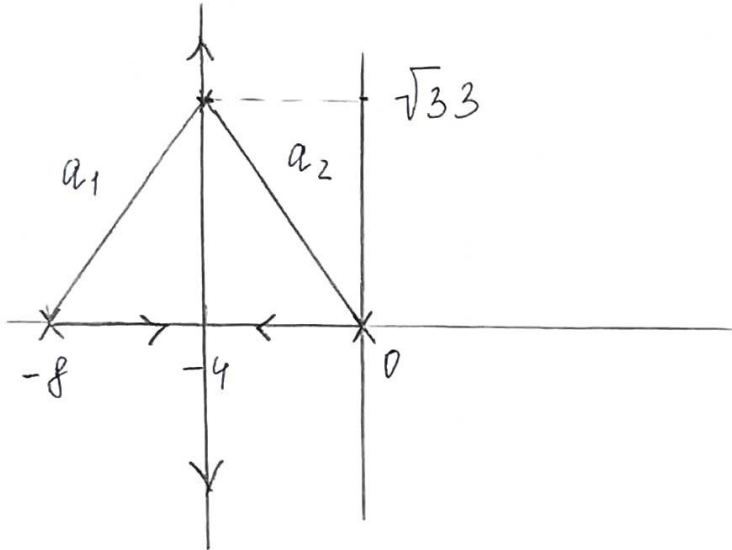
If  $\omega_n = 7$  and  $\zeta \omega_n = 4$ , then  $\zeta = \frac{4}{7} \approx 0.571$

Therefore, all requirements are satisfied if the

test point is:  $s^* = -4 + \sqrt{33}j$

To find  $K$ , magnitude condition is used:

$$\left| \frac{N(s)}{D(s)} \right|_{s^*} = \left| -\frac{1}{K} \right| = \frac{1}{K}$$



$$\frac{1}{a_1 a_2} = \frac{1}{K}$$

$$a_1 = a_2 = \sqrt{4^2 + 33} = 7$$

$$\frac{1}{7 \cdot 7} = \frac{1}{K} \Rightarrow \boxed{K = 49}$$

$$\Rightarrow G_c(s) = 49$$

MATLAB verification is done with the help of  $rltool(Gp)$ .

Design requirements are set in Root Locus Editor window (Figure 1).

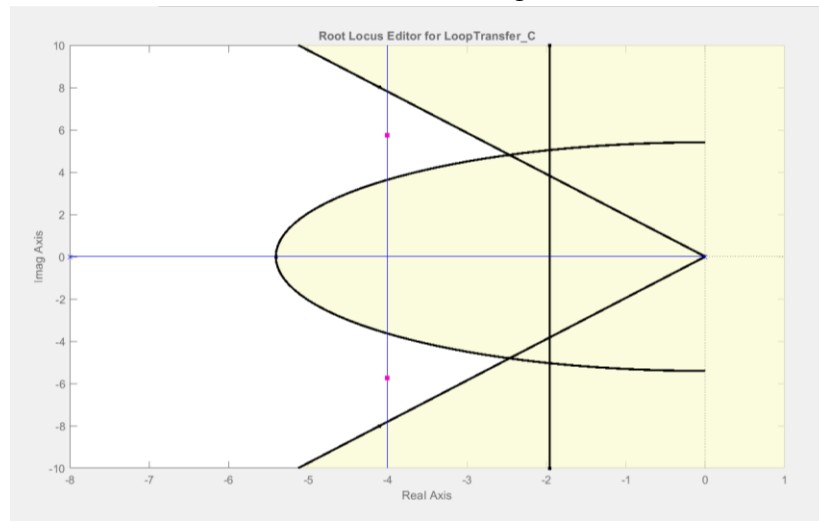


Figure 1: Setting Design Requirements

Equating the gain of a compensator to 49, we obtain the following step response and its parameters (Figure 2).

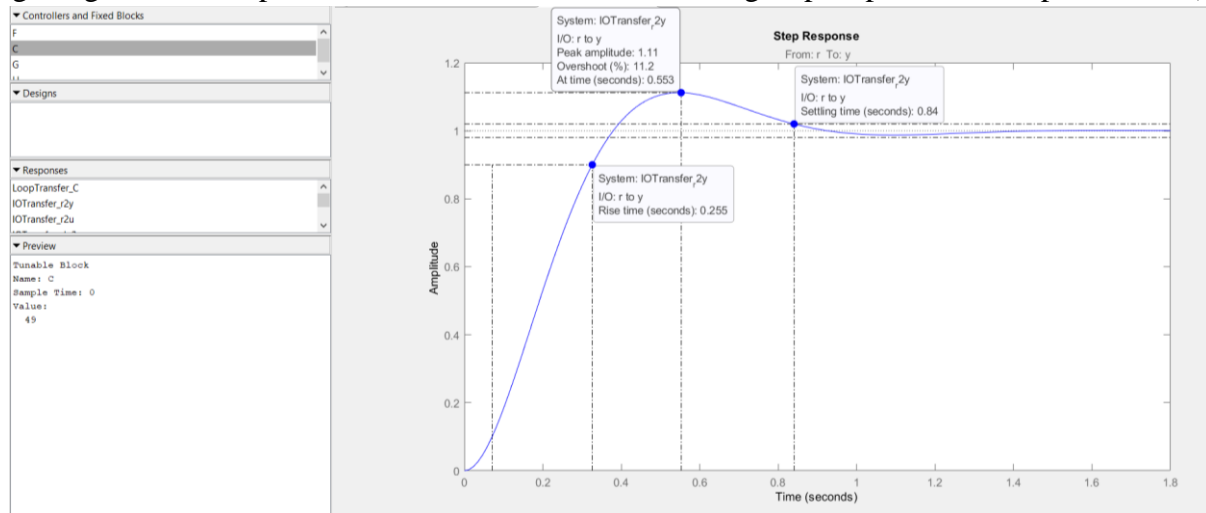


Figure 2: Step Response and Its Parameters for  $G_c(s) = 49$

It is seen from Figure 2 that all design requirements are satisfied, therefore, the design is verified.

$$2. \quad G_p(s) = \frac{10}{s^2}$$

Design a practically realizable compensator  $G_c(s)$  to locate the dominant poles of the closed loop system at

$$s = -1 \pm 1.3j$$

$$L(s) = G_c(s) \cdot G_p(s) = 10K \frac{1}{s^2}$$

$$N(s) = 1$$

$$\Rightarrow \begin{aligned} n &= 2 \\ m &= 0 \end{aligned}$$

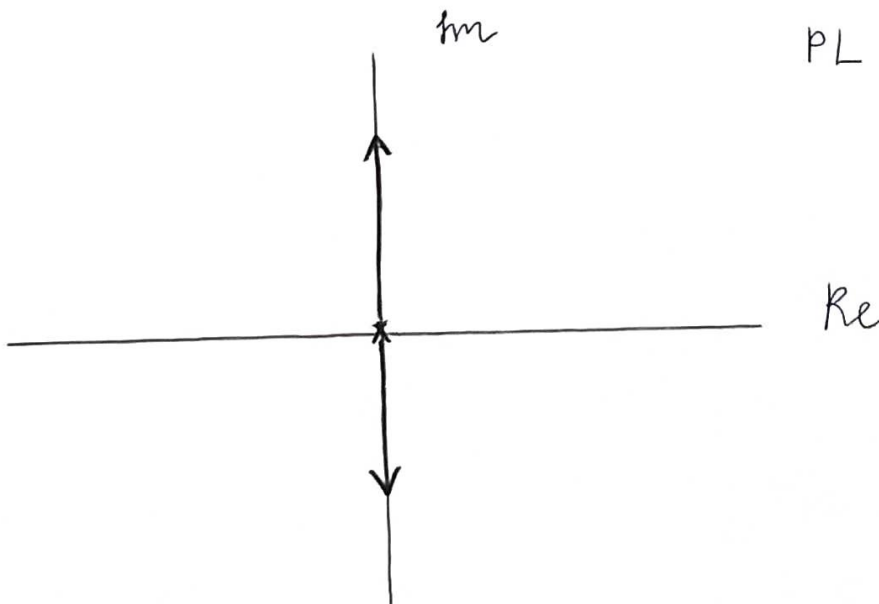
$$D(s) = s^2$$

$$r = 2 \Rightarrow 2 \text{ asymptotes}$$

$$\sigma_a = 0$$

$$\psi_a = \frac{\pi}{2}, \frac{3\pi}{2} \quad (PL)$$

No zeros, Poles: 2 poles at 0,



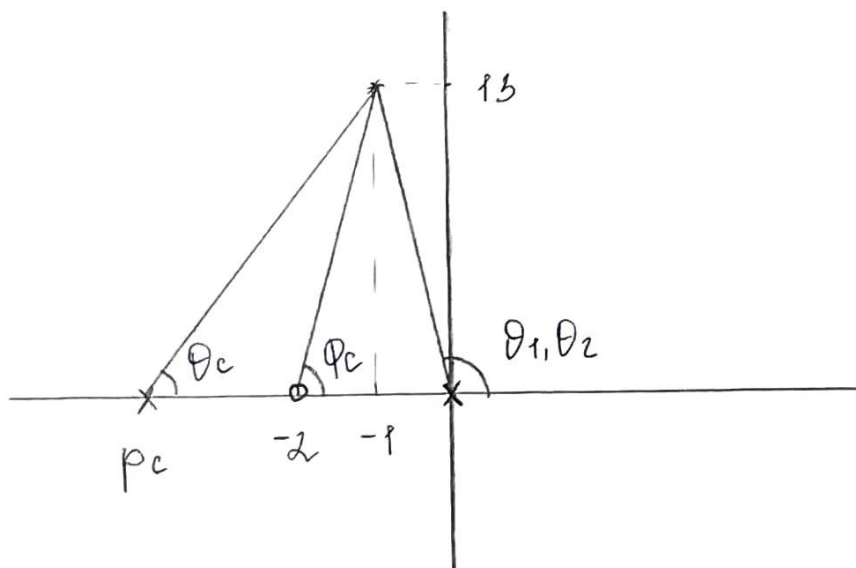
To have the dominant poles at  $s = -1 \pm 13j$ , we need to change the root locus plot,

This could be done with a lead compensator:

$$G_c(s) = K \frac{s + z_c}{s + p_c} \quad (\text{practically realizable})$$

Choose arbitrarily:  $z_c = -2$

Find  $p_c$  using the phase condition



$$\theta_1 = \theta_2 = 180^\circ - \tan^{-1}(13) = 94.4^\circ$$

$$\phi_c = \tan^{-1}(13) = 85.6^\circ$$

$$\phi_c - (\theta_c + \theta_1 + \theta_2) = -180^\circ$$

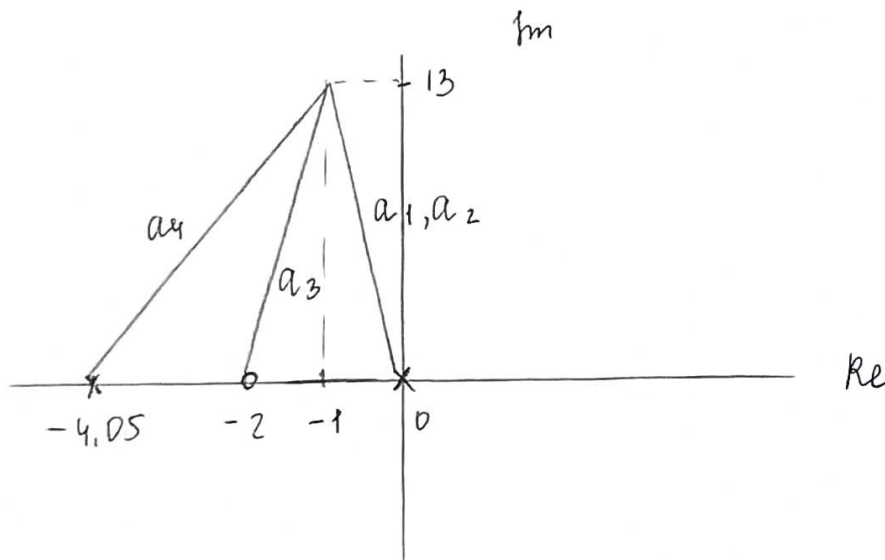
$$\theta_c = 180^\circ + \phi_c - \theta_1 - \theta_2 = 180^\circ + 85.6^\circ - 2 \cdot 94.4^\circ = 76.8^\circ$$

$$\tan(76.8^\circ) = \frac{13}{p_c - 1}$$

$$p_c = 4.05$$

To find  $K$ , use the magnitude condition:

$$\left| \frac{N(s)}{D(s)} \right|_{s^*} = \left| -\frac{1}{P} \right| = \frac{1}{10K}$$



$$a_1 = a_2 = a_3$$

$$\frac{a_3}{a_1 a_2 a_4} = \frac{1}{10K}$$

$$a_1 = \sqrt{13^2 + 1^2} = \sqrt{170}$$

$$a_4 = \sqrt{13^2 + 3.05^2} \approx 13.35$$

$$\frac{1}{\sqrt{170} \cdot 13.35} = \frac{1}{10K}$$

$$\Rightarrow K = 17.4$$

$\Rightarrow$

$$G_c(s) = 17.4 \frac{(s+2)}{(s+4.05)}$$



$$\Rightarrow L(s) = G_c(s) \cdot G_p(s) = 17.4 \frac{(s+2)}{(s+4.05)} \cdot \frac{10}{s^2} =$$

$$\frac{174 (s+2)}{s^2 (s+4.05)}$$

$$\Rightarrow T(s) = \frac{L(s)}{1+L(s)} = \frac{174 (s+2)}{s^2 (s+4.05) + 174(s+2)} =$$

$$\frac{174 (s+2)}{s^3 + 4.05 s^2 + 174 s + 348}$$

$$s^3 + 4.05 s^2 + 174 s + 348 = 0$$

$$s_1 \approx -2.048 \quad s_{2,3} = -1.001 \pm 12.996j'$$

Therefore, the dominant poles are indeed at  $s = -1 \pm 13j$  (approximately)

As the third pole and zero almost coincide, the second-order approximation is valid

The second-order approximation:

$$T(s) = \frac{174}{s^2 + 2s + 170}$$

$$\omega_n^2 = 170$$

$$2\zeta\omega_n = 2$$

$$T_r = \frac{1.8}{\omega_n} = \frac{1.8}{\sqrt{170}} \approx 0.138 \text{ s}$$

$$T_s = \frac{4}{\zeta\omega_n} = \frac{4}{1} = 4 \text{ s}$$

$$\begin{aligned} M_p &= 100 \cdot e^{-\zeta\pi / \sqrt{1-\zeta^2}} = 100e^{-\frac{1}{\sqrt{170}}\pi / \sqrt{1-\frac{1}{170}}} \\ &\approx 78.5\% \end{aligned}$$

MATLAB code:

```
L = zpk(-2, [0 0 -4.05], 174)
T = feedback(L, 1)
figure
step(T)
stepinfo(T)
```

The transfer function of the open loop-system is defined using *zpk()*, and the closed-loop system is defined using *feedback()*. Then, step response of the closed-loop system is plotted (Figure 3), and its parameters are measured using *stepinfo()* (Figure 4).

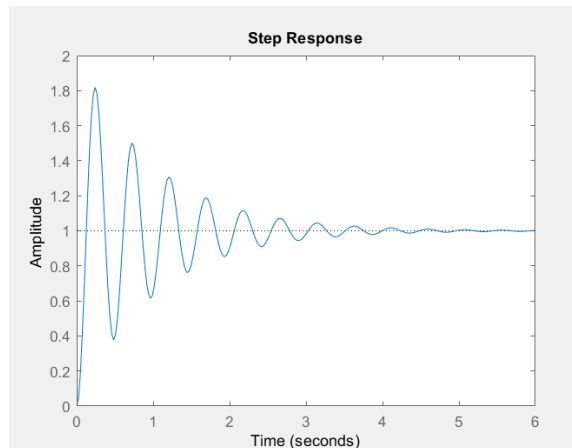


Figure 3: Step Response of the Closed-Loop System

`struct` with fields:

```
RiseTime: 0.0836
SettlingTime: 3.8923
SettlingMin: 0.3778
SettlingMax: 1.8188
Overshoot: 81.8751
Undershoot: 0
Peak: 1.8188
PeakTime: 0.2410
```

Figure 4: Parameters of Step Response

Comparing the calculated values of the parameters with those obtained via MATLAB, it is seen the percent overshoot and settling time values are almost the same in both cases. However, the rising time values differ more, therefore, the approximation of rising time is not good enough.

$$3. \quad G_p(s) = \frac{1}{s(s+2)(s+4)}$$

Requirements:  $T_s < 3s$

$$M_p < 20\%$$

$$e_{ss} < 0.1 \quad \text{to a unit ramp}$$

$$L(s) = G_c(s) G_p(s) = K \frac{1}{s(s+2)(s+4)}$$

$$N(s) = 1$$

$$n = 3$$

$$m = 0$$

$$D(s) = s(s+2)(s+4)$$

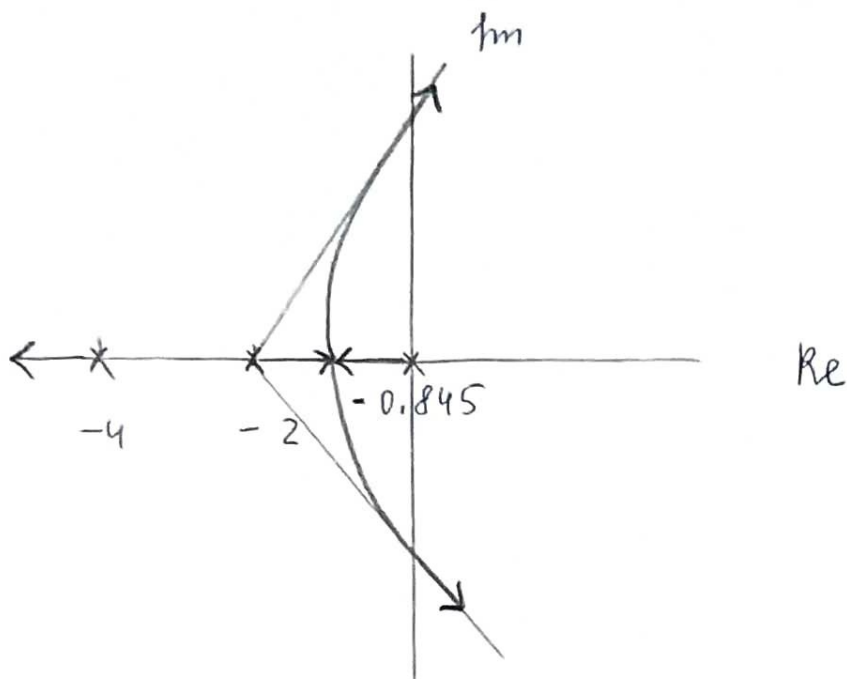
$$r = 3 \Rightarrow 3 \text{ asymptotes}$$

$$\sigma_a = \frac{0 - (0+2+4)}{3} = -2$$

$$3 \psi_q = (2h+1)\pi \quad h=0,1,2 \quad (PL)$$

$$\psi_a = \frac{\pi}{3}, \pi, \frac{5\pi}{3}$$

|            |                 |         |          |
|------------|-----------------|---------|----------|
| Real axis: | $(-\infty, -4)$ | 3 poles | $\in PL$ |
|            | $(-4, -2)$      | 2 poles |          |
|            | $(-2, 0)$       | 1 pole  | $\in PL$ |
|            | $(0, +\infty)$  | 0 poles |          |



Break in/away points:

$$N(s)D'(s) - N'(s)D(s) = 0 \quad D(s) = s^3 + 6s^2 + 8s$$

$$3s^2 + 12s + 8 = 0$$

$$s_1 = -0.845 \quad s_2 = -3.155$$

$$T_s = \frac{4}{\zeta \omega_n} < 3$$

$$\zeta \omega_n > \frac{4}{3} \approx 1.333$$

$$\bar{\zeta} = \frac{-\ln(\bar{\sigma}_{lp}/100)}{\sqrt{\pi^2 + \ln^2(\bar{\sigma}_{lp}/100)}} = -\frac{\ln(20/100)}{\sqrt{\pi^2 + \ln^2(20/100)}} \approx 0.456$$

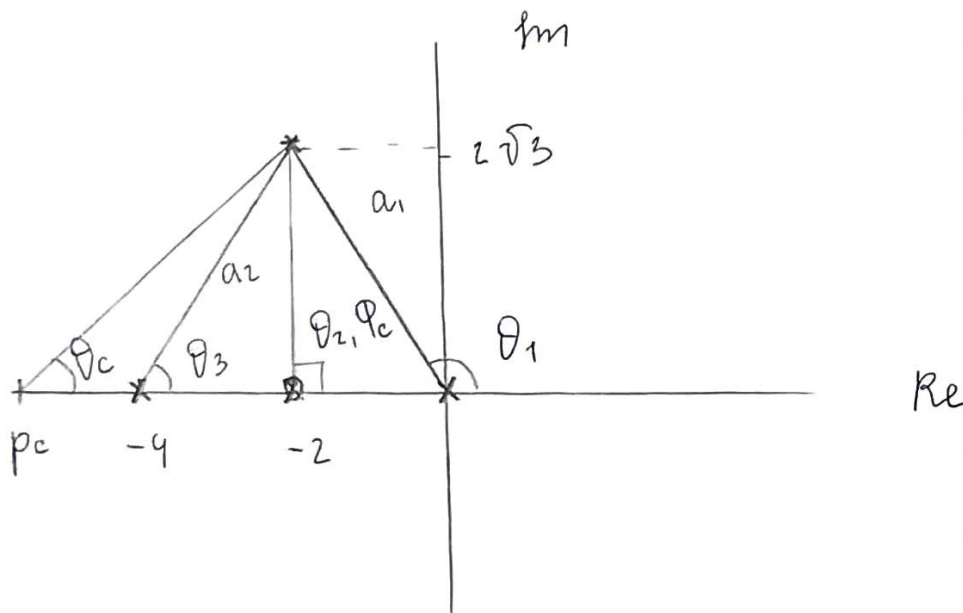
$$\Rightarrow \zeta > 0.456$$

Choose  $\zeta = 0.5$ ,  $\omega_n = 2$   
( $\theta = 60^\circ$ )

$\Rightarrow$  A test point is:  $s^* = -2 + 2\sqrt{3}j$

Choose arbitrarily:  $z_c = 2$

Find  $p_c$  using the phase condition:



$\phi_c$  and  $\theta_2$  cancel each other

$$\theta_1 = 180 - \tan^{-1}\left(\frac{2\sqrt{3}}{2}\right) = 120^\circ$$

$$\theta_3 = \tan^{-1}\left(\frac{2\sqrt{3}}{2}\right) = 60^\circ$$

$$-\theta_1 - \theta_3 - \phi_c = -180^\circ$$

$$\phi_c = 180^\circ - 120^\circ - 60^\circ = 0$$

$\Rightarrow p_c$  is at infinity and can be neglected

To find  $K$ , use the magnitude condition

$$\left| \frac{N(s)}{D(s)} \right|_{s^*} = \left| -\frac{1}{K} \right| = \frac{1}{K}$$

$$\frac{1}{a_1 \cdot a_2} = \frac{1}{K}$$

$$a_1 = a_2 = \sqrt{2^2 + (2\sqrt{3})^2} = 4$$

$$\frac{1}{4 \cdot 4} = \frac{1}{K} \Rightarrow K = 16$$

$$L(s) = G_c(s) G_p(s) = 16(s+2) \cdot \frac{1}{s(s+2)(s+4)}$$

$$K_v = \lim_{s \rightarrow 0} s L(s) = 4$$

$$e_{ss} = \frac{1}{K_v} = \frac{1}{4} = 0.25 \quad \text{to a unit ramp}$$

$\Rightarrow$  steady-state error requirement is not satisfied

But introducing a PI controller with arbitrarily chosen zero:  $z_c = -0.01$ ,

we increase the system's type  $\Rightarrow$

$$e_{ss} = 0$$

$$\Rightarrow G_c(s) = \frac{16(s+2)(s+0.01)}{s}$$

Design requirements are set in Root Locus Editor window (Figure 5).



Figure 6: Step Response and Its Parameters

It is seen from Figure 5 that percent overshoot and settling time requirements are satisfied. The steady-state error requirement for a unit ramp is also satisfied because the open-loop system is a Type-2 system. Therefore, the design is verified.



$$4. \quad G_p(s) = \frac{1600}{(s+2)(s+4)(s^2+s+200)}$$

Requirements:  $e_{ss} < 0.1$  for a unit step

$$T_{s, 2\%} < 2s$$

$$\text{M}_p < 20\%$$

$$L(s) = G_c(s) G_p(s) = K \frac{1600}{(s+2)(s+4)(s^2+s+200)}$$

$$N(s) = 1$$

$$n = 4$$

$$D(s) = (s+2)(s+4)(s^2+s+200)$$

$$m = 0$$

$$r = 4 \Rightarrow 4 \text{ asymptotes}$$

No zeros

$$\text{Poles: } (s+2)(s+4)(s^2+s+200) = 0$$

$$s_1 = -2$$

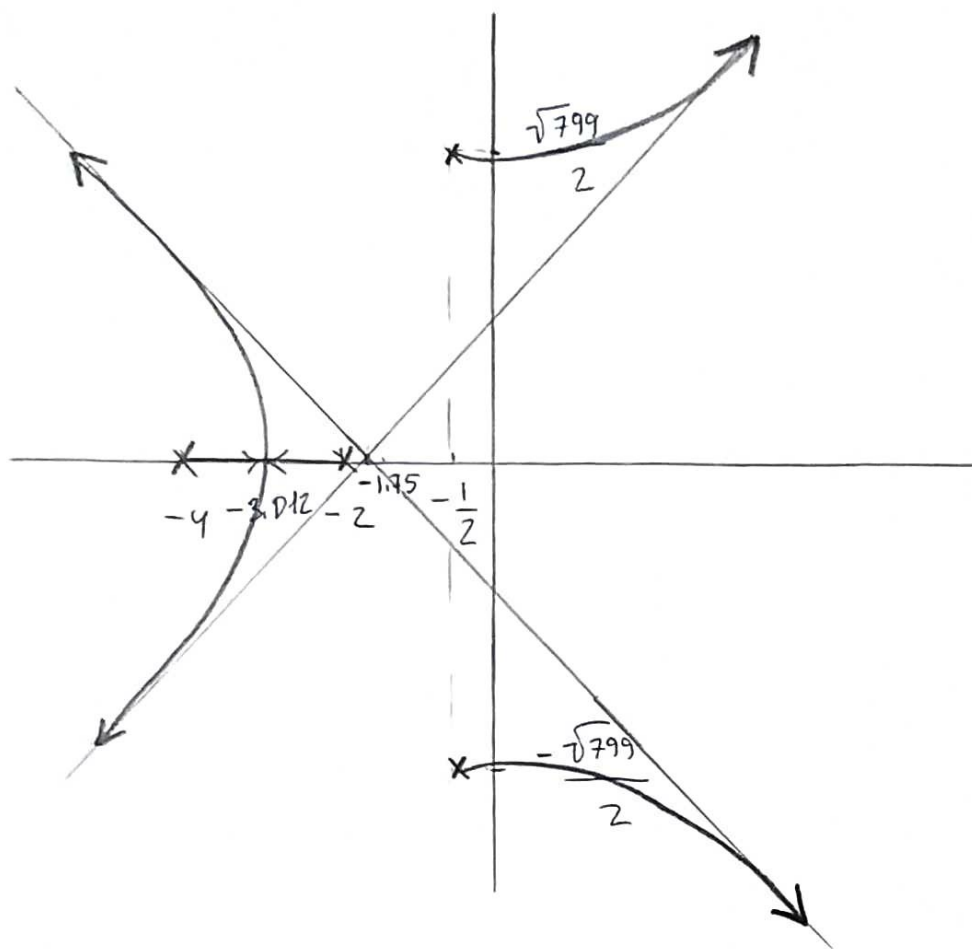
$$s_2 = -4$$

$$s_{3,4} = -\frac{1}{2} \pm \frac{\sqrt{799}}{2} j$$

$$\sigma_a = \frac{0 - \left(2 + 4 + \frac{1}{2} + \frac{1}{2}\right)}{4} = -\frac{7}{4} = -1.75$$

$$4\psi_a = (2h+1)\pi \quad h = 0, 1, 2, 3 \quad (PL)$$

$$\psi_a = \frac{\pi}{4}, \frac{3\pi}{4}, \frac{5\pi}{4}, \frac{7\pi}{4} \quad (PL)$$



|            |                           |         |          |
|------------|---------------------------|---------|----------|
| Real axis: | $(-\infty, -4)$           | 4 poles | $\in NL$ |
|            | $(-4, -2)$                | 3 poles | $\in PL$ |
|            | $(-2, -\frac{1}{2})$      | 2 poles | $\in NL$ |
|            | $(-\frac{1}{2}, +\infty)$ | 0 poles | $\in NL$ |

Break-in/away:

$$N(s)D'(s) - N'(s)D(s) = 0 \quad D(s) = s^4 + 7s^3 + 214s^2 + 1208s + 1600$$

$$4s^3 + 21s^2 + 428s + 1208 = 0$$

$$s \approx -3.012$$

The requirements can be met if a notch filter is used.

The zeros of the notch filter must be placed close to the high-frequency poles:  $-\frac{1}{2} \pm \frac{\sqrt{799}}{2} j \approx -\frac{1}{2} \pm 14.13j$

$\Rightarrow$  Place zeros at  $-\frac{1}{2} \pm 14.2j$

Also, two poles must be placed on the real axis, far from the origin. Choose:  $-50$  and  $-55$

Therefore, the notch filter is:  $G_c(s) = \frac{K(s^2 + s + 201.89)}{(s+50)(s+55)}$

To find  $K$ , hard computations are needed  
Therefore, the help of Matlab `rltool()` is used

Design requirements are set in Root Locus Editor window (Figure 7). The closed-loop dominant complex poles must be located in the white area. As a result, the obtained gain is 54200 (Figure 7).

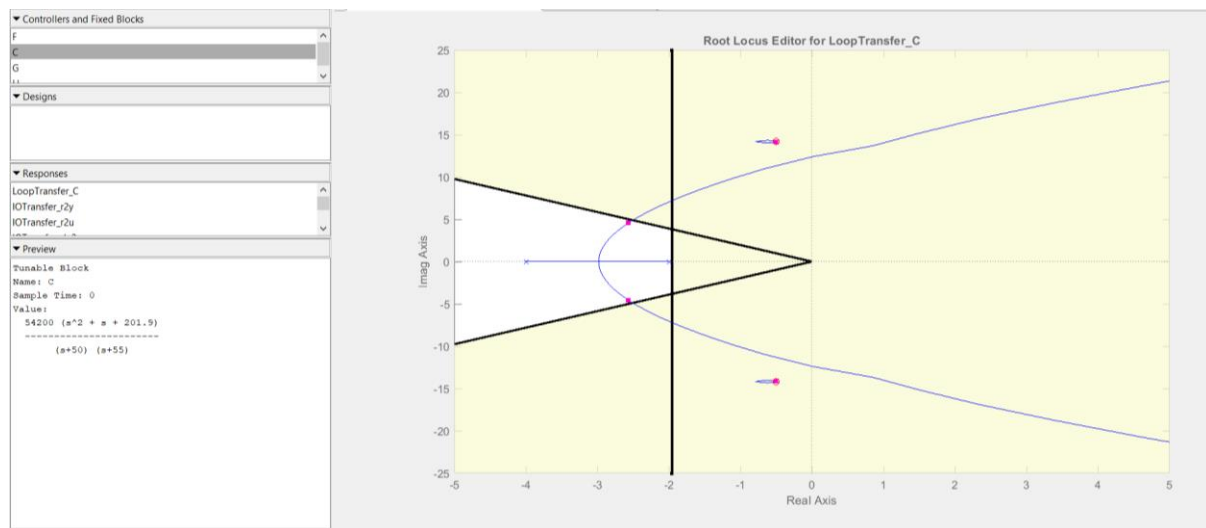


Figure 7: Setting Design Requirements and Obtaining Gain

With the obtained gain, the percent overshoot and settling time requirements are satisfied, whereas the steady-state error requirement is not (Figure 8).

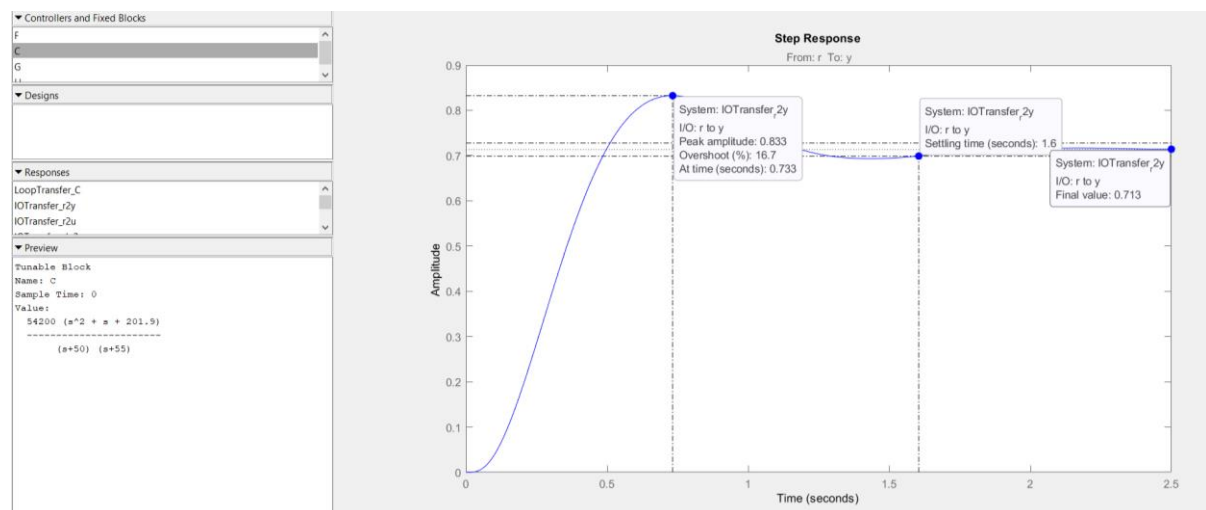


Figure 8: Step Response and Its Parameters

The steady-state error requirement can be satisfied by using a PI integrator with a zero placed close to the origin:  $G_c(s) = \frac{s+0.01}{s}$   
(system's type is increased to Type-1)

$\Rightarrow$  The designed compensator is:

$$G_c(s) = \frac{54200 (s+0.01) (s^2+s+201.89)}{s (s+50) (s+55)}$$