Question 1: Delta Distribution

Evaluate the following integrals

(a)

$$\int_0^\infty \delta(2t^{1/3} - 54)e^{2t}\mathrm{d}t$$

Hint: Make a variable substitution $u=2t^{1/3}$ to bring the Delta function into a standard form.

Let $u=2t^{1/3}$ and thus $t=rac{1}{8}u^3$, and thus

$$\mathrm{d}t = \frac{3}{8}u^2\mathrm{d}u$$

and thus

$$\int_0^\infty \delta(u - 54)e^{\frac{1}{4}u^3} \frac{3}{8}u^2 du = \frac{2187}{2}e^{39366}$$

(b)

$$\int_{-\infty}^{\infty} \left(\frac{\mathrm{d}^2}{\mathrm{d}t^2} \delta(t-4) \right) \sin\left(\frac{t^2}{4}\right) \mathrm{d}t$$

Hint: you may integrate by parts

We could actually find a more general rule, notice that $\delta^{(n)}(\pm \infty)=0$, and that means

$$\int_{-\infty}^{\infty} f(x) \delta^{(n)}(x) \mathrm{d}x = f(x) \delta^{(n)}(x)|_{-\infty}^{\infty} - \int_{-\infty}^{\infty} \frac{\mathrm{d}}{\mathrm{d}x} f(x) \delta^{(n-1)} \mathrm{d}x = - \int_{-\infty}^{\infty} \frac{\mathrm{d}}{\mathrm{d}x} f(x) \delta^{(n-1)} \mathrm{d}x$$

and thus

$$\int_{-\infty}^{\infty} f(x)\delta^{(n)}(x)\mathrm{d}x = (-1)^n \int_{-\infty}^{\infty} \frac{\mathrm{d}^n}{\mathrm{d}t^n} f(x)\delta(x)\mathrm{d}x$$

and thus

$$\int_{-\infty}^{\infty} \left(\frac{\mathrm{d}^2}{\mathrm{d}t^2} \delta(t - 4)\right) \sin\left(\frac{t^2}{4}\right) \mathrm{d}t$$

$$= \int_{-\infty}^{\infty} \delta(t - 4) \frac{\mathrm{d}^2}{\mathrm{d}t^2} \sin\left(\frac{t^2}{4}\right) \mathrm{d}t$$

$$= \int_{-\infty}^{\infty} \delta(t - 4) \left(\frac{1}{2} \cos\left(\frac{t^2}{4}\right) - \frac{1}{4} t^2 \sin\left(\frac{t^2}{4}\right)\right) \mathrm{d}t$$

$$= \frac{1}{2} \cos(4) - 4 \sin(4)$$

Question 2: Impulse response

Consider a damped oscillator with impulse response, i.e., Green's function, given by

$$G(t) = \begin{cases} 2e^{-t}\sin(4t) & t > 0\\ 0 & t < 0 \end{cases}$$

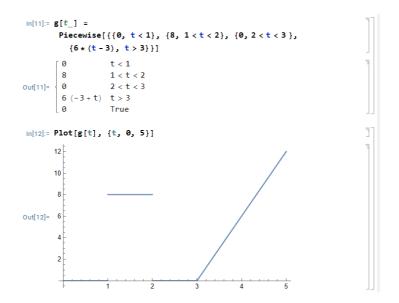
and subjected to a force

$$F(t) = \begin{cases} 0 & t < 1 \\ 8 & 1 < t < 2 \\ 0 & 2 < t < 3 \\ 6(t-3) & t > 3 \end{cases}$$

(a)

Sketch the force

By Mathematica



(b)

Construct an integral convolution expression for the response that is valid at all times. You do not need to evaluate the integrals, but they should be of a form that one could look up in a table of integrals.

the convolution have the form of

$$x(t) = \int_{-\infty}^t F(au) G(t- au) \mathrm{d} au$$

for t < 1, the F(au) = 0 in above expression, and thus

$$x(t) = \int_{-\infty}^{t} F(\tau)G(t-\tau)\mathrm{d} au = 0$$

for 1 < t < 2,

$$x(t) = \int_{-\infty}^t F(au)G(t- au)\mathrm{d} au = \int_1^t F(au)G(t- au)\mathrm{d} au = \int_1^t 8\cdot 2e^{-(t- au)}\sin(4(t- au))\mathrm{d} au$$

for 2 < t < 3, the F(au) = 0 for 2 < au < 3

$$x(t) = \int_{-\infty}^t F(au) G(t- au) \mathrm{d} au = \int_1^2 8 \cdot 2e^{-(t- au)} \sin(4(t- au)) \mathrm{d} au$$

for t > 3,

$$x(t) = \int_{-\infty}^{t} F(\tau)G(t-\tau)d\tau = \int_{1}^{2} 8 \cdot 2e^{-(t-\tau)} \sin(4(t-\tau))d\tau + \int_{3}^{t} 6(\tau-3)2e^{-(t-\tau)} \sin(4(t-\tau))d\tau$$

and the integral could be evaluated.

Question 3: Piecewise driving force

A force acts on an initial quiescent ($x_0 = x(t=0) = 0, v_0 = v(t=0) = 0$), undamped mass-spring system described by the equation of motion

$$m\frac{\mathrm{d}^2x}{\mathrm{d}t^2} + kx = F(t)$$

where the driving force F(t) is piece-wise analytic

$$F(t) = egin{cases} 0 & t < 0 \ F_0 rac{t^2}{T^2} & 0 < t < T \ F_0 & T < t \end{cases}$$

Like F(t), the response x(t) will be piece-wise analytic, i.e., it has different analytic form depending on whether t is smaller or larger than T.

In both cases, you do not need to evaluate the integral, but you should conclude with an expression that one could look up in a table of integrals.

From (41) of Lecture 16, we know that since the equation is undamped

$$G(t) = egin{cases} rac{\sin(\omega_d t)}{m \omega_d} & & t > 0 \ 0 & & t < 0 \end{cases}$$

(a)

Use the convolution to compose an expression for the response x(t) at times 0 < t < T.

$$x(t) = \int_{-\infty}^{t} F(\tau)G(t-\tau)d\tau$$

$$= \int_{0}^{t} F(\tau)G(t-\tau)d\tau$$

$$= \int_{0}^{t} F_{0}\frac{t^{2}}{T^{2}}\frac{\sin(\omega_{d}(t-\tau))}{m\omega_{d}}d\tau$$

$$= \frac{F_{0}}{m\omega_{d}}\frac{t^{2}}{T^{2}}\int_{0}^{t}\sin(\omega_{d}(t-\tau))d\tau$$

(b)

Use the convolution to compose an expression for the response x(t) valid at times t>T

$$x(t) = \int_{-\infty}^{t} F(\tau)G(t-\tau)d\tau$$

$$= \int_{0}^{T} F(\tau)G(t-\tau)d\tau + \int_{T}^{t} F(\tau)G(t-\tau)d\tau$$

$$= \int_{0}^{T} F_{0}\frac{t^{2}}{T^{2}}\frac{\sin(\omega_{d}(t-\tau))}{m\omega_{d}}d\tau + F_{0}\int_{T}^{t}\frac{\sin(\omega_{d}(t-\tau))}{m\omega_{d}}d\tau$$

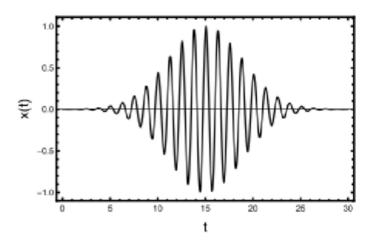
$$= \frac{F_{0}}{m\omega_{d}}\left(\int_{0}^{T} \frac{t^{2}}{T^{2}}\sin(\omega_{d}(t-\tau))d\tau + \int_{T}^{t}\sin(\omega_{d}(t-\tau))d\tau\right)$$

Question 4: Gaussian tone burst

The complex Gaussian tone burst

$$x(t) = e^{i\alpha t} \exp\left(-\frac{(t - t_0)^2}{T^2}\right)$$

is oscillatory with nominal frequency α , under a Gaussian envelop of width T and centered on time t_0 . The figure plots its real part for the case $\alpha=5, T=\sqrt{30}$ and $t_0=15$. If this was a plot of pressure on your ear versus time, you would hear a short beep, assuming that the frequency $f=\frac{\alpha}{2\pi}$ is within the hearing range.



(a)

Show that

$$| ilde{x}(\omega)| = \sqrt{\pi}T\expigg(-lpharac{(lpha-\omega)^2T^2}{4}igg)$$

Hint: you may use

$$\int_{-\infty}^{\infty} \exp(i\beta x) \exp(-x^{2}) dx = \sqrt{\pi} \exp\left(-\frac{\beta^{2}}{4}\right)$$

$$\tilde{x}(\omega) = \int_{-\infty}^{\infty} x(t) \exp(-i\omega t) dt = \int_{-\infty}^{\infty} e^{i\alpha t} \exp\left(-\frac{(t-t_{0})^{2}}{T^{2}}\right) \exp(-i\omega t) dt$$

$$= \int_{-\infty}^{\infty} \exp(i(\alpha - \omega)t) \exp\left(-\left(\frac{t-t_{0}}{T}\right)^{2}\right) dt$$

$$= \int_{-\infty}^{\infty} \exp(i(\alpha - \omega)(T\tau + t_{0})) \exp(-\tau^{2}) T d\tau$$

$$= T \exp(i(\alpha - \omega)t_{0}) \int_{-\infty}^{\infty} \exp(i(\alpha - \omega)T\tau) \exp(-\tau^{2}) d\tau$$

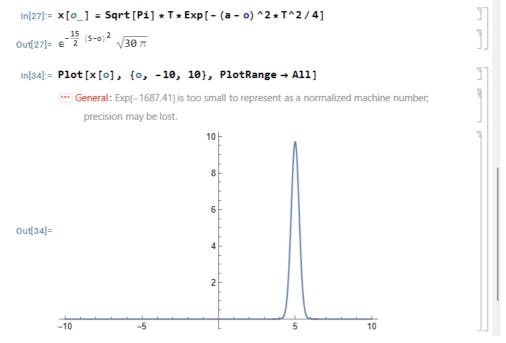
$$= T \exp(i(\alpha - \omega)t_{0}) \sqrt{\pi} \exp\left(-\frac{(\alpha - \omega)^{2}T^{2}}{4}\right)$$

$$|\tilde{x}(\omega)| = T\sqrt{\pi} \exp\left(-\frac{(\alpha - \omega)^{2}T^{2}}{4}\right)$$

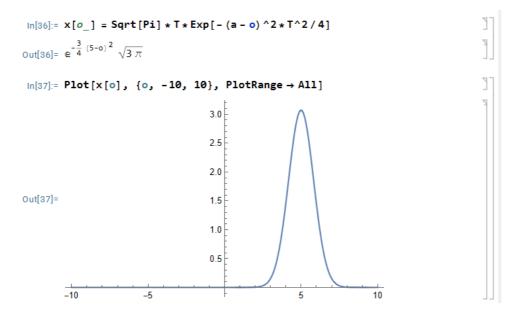
(b)

Plot the absolute value $|\tilde{x}(\omega)|$ for two cases (i) $T=30, \alpha=5, t_0=15$, and (ii) $T=\sqrt{3}, \alpha=5, t_0=15$

when
$$T=\sqrt{30}, lpha=5, t_0=15$$



when $T=\sqrt{3}, \alpha=5, t_0=15$,



(c)

Show that $|\tilde{x}(\omega)|$ peaks at the nominal frequency $\omega=\alpha$. Discuss how your result illustrates that large T (i.e., being well spread out in time) corresponds to a Fourier transform that is highly compact in frequency.

From the graph, we could indeed see that it peek at $\alpha=5$. The reason is simply because $(a-\omega)^2$ has its minimum value at $\alpha=\omega$, and thus $\exp\left(-\frac{(\alpha-\omega)^2T^2}{4}\right)$ has maximum value and thus $|\tilde{x}(\omega)|$.

The graph illustrate that larger T result in a more compact peak. This is expected since with higher T, $-\frac{(\alpha-\omega)^2T^2}{4}$ decrease more rapidly when away from the $\omega=\alpha$, and thus the $|\tilde{x}(\omega)|$. A more intuitive explanation comes from signal processing. If you have a sound at certain frequency for a longer period of time, you are more certain about the frequency of that sound (which, when illustrate on the plot of $|\tilde{x}(\omega)|$, is a more compact peak).