

## Lab 3. Simulation components of dynamic systems

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Specialization: Automation

Variant	U	$\Psi$	R	L	J_M	J_L	b	k
94	24	$0.22918_3$	0.35	0.00035	$0.00150_1$	$0.00450_2$	0.06	1700

### 1. Simscape model of DC-motor.

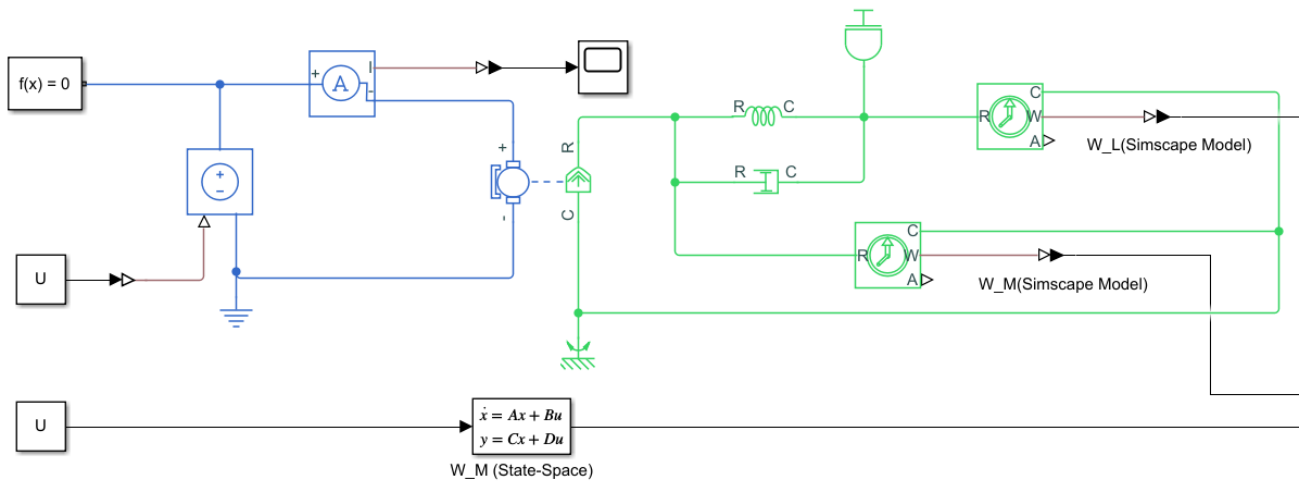


Figure 1. Equivalent circuit.

## 2. Block diagram model of DC-motor.

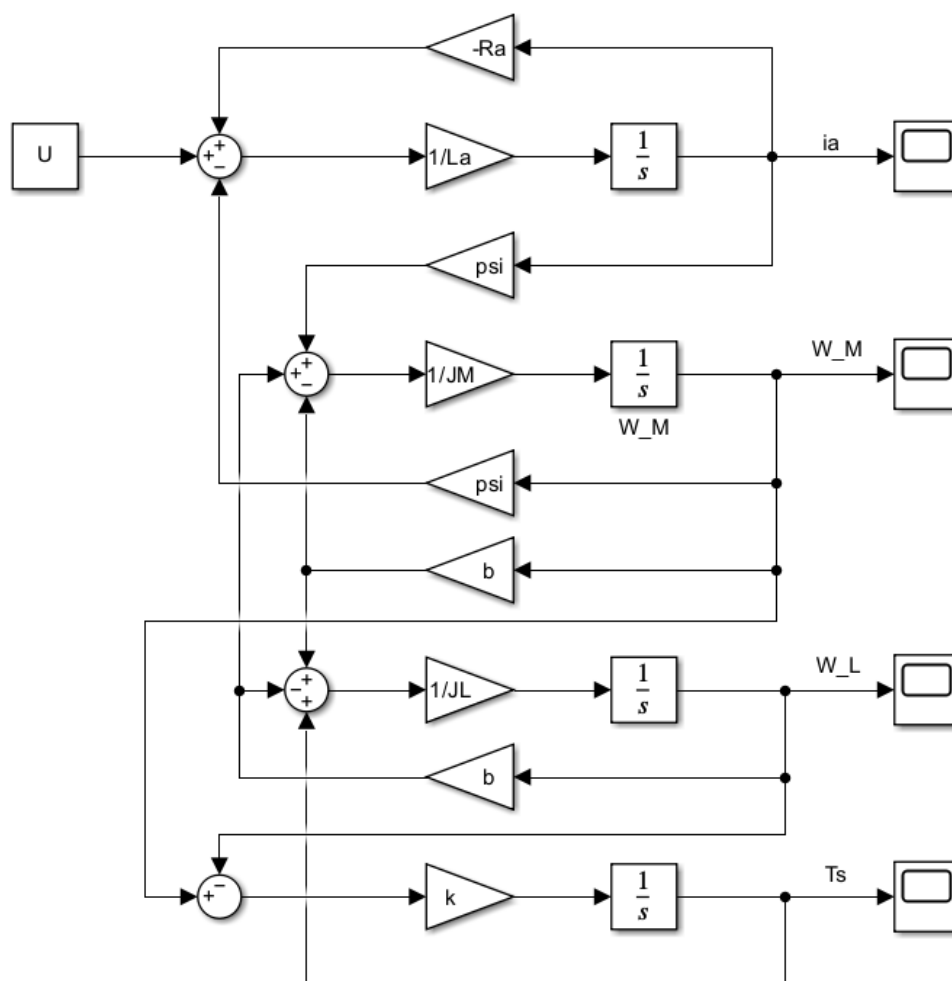


Figure 2. Simulation circuit.

## 2. Transfer functions of DC-motor.

$$W_1 = \frac{\omega_M}{U} = \frac{\psi(J_L \cdot s^2 + b \cdot s + k)}{J_L J_M L_a \cdot s^4 + (J_L J_M R_a + J_L L_a b + J_M L_a b) \cdot s^3 + (J_L \psi^2 + J_L R_a b + J_M R_a b + J_L L_a k + J_M L_a k) \cdot s^2 + (\psi^2 b + J_L R_a k + J_M R_a k) \cdot s + \psi^2 k}$$

$$W_2 = \frac{\omega_L}{U} = \frac{\psi b \cdot s + \psi k}{J_L J_M L_a \cdot s^4 + (J_L J_M R_a + J_L L_a b + J_M L_a b) \cdot s^3 + (J_L \psi^2 + J_L R_a b + J_M R_a b + J_L L_a k + J_M L_a k) \cdot s^2 + (\psi^2 b + J_L R_a k + J_M R_a k) \cdot s + \psi^2 k}$$

### 3. State-space model.

$$L_a \cdot \frac{di_a}{dt} = U - R_a \cdot i_a - \Psi \cdot \omega_M$$

$$J_M \cdot \frac{d\omega_M}{dt} = \Psi \cdot i_a - T_S - b(\omega_M - \omega_L)$$

$$J_L \cdot \frac{d\omega_L}{dt} = T_S + b(\omega_M - \omega_L)$$

$$\frac{dT_S}{dt} = k(\omega_M - \omega_L)$$

$$A = \begin{bmatrix} -\frac{R_a}{L_a} & -\frac{\psi}{L_a} & 0 & 0 \\ \frac{\psi}{J_M} & -\frac{b}{J_M} & \frac{b}{J_M} & -\frac{1}{J_M} \\ 0 & \frac{b}{J_L} & -\frac{b}{J_L} & \frac{1}{J_L} \\ 0 & k & -k & 0 \end{bmatrix} \quad B = \begin{bmatrix} \frac{1}{L_a} \\ 0 \\ 0 \\ 0 \end{bmatrix} \quad C = \begin{bmatrix} 0 & 1 & 0 & 0 \end{bmatrix} \quad D = 0$$

### 5. Simulation results for 2 cases

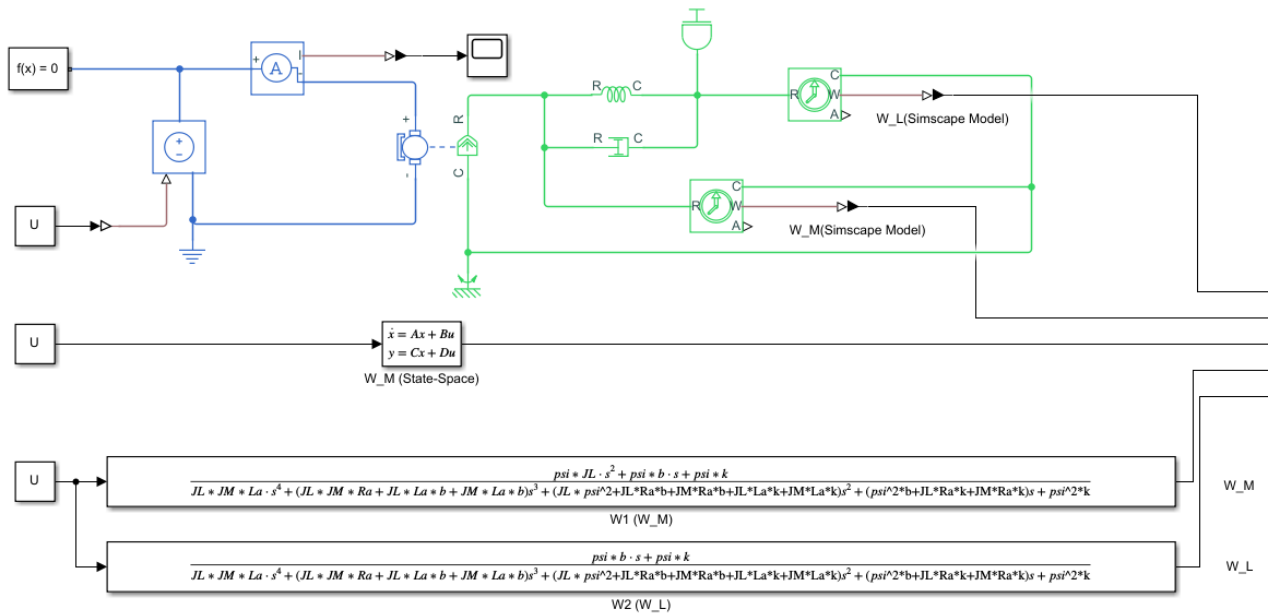
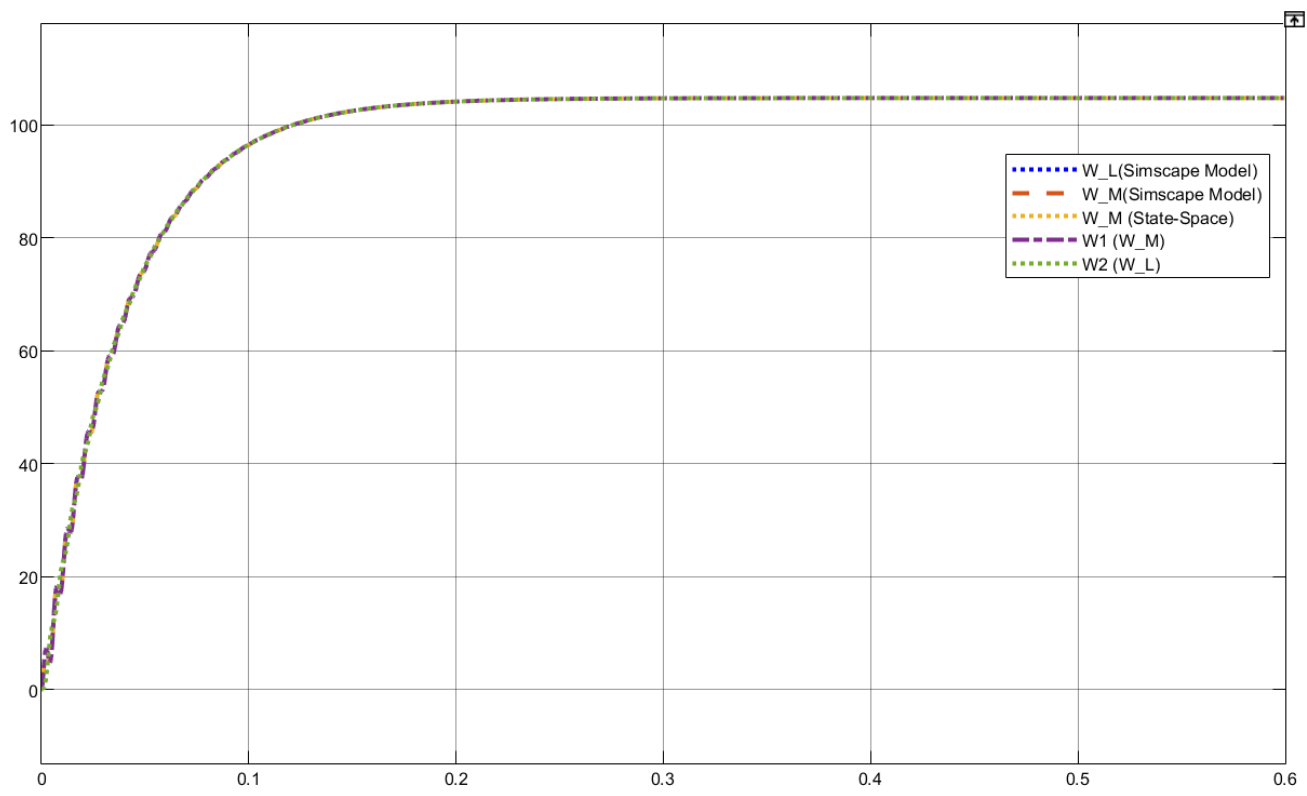
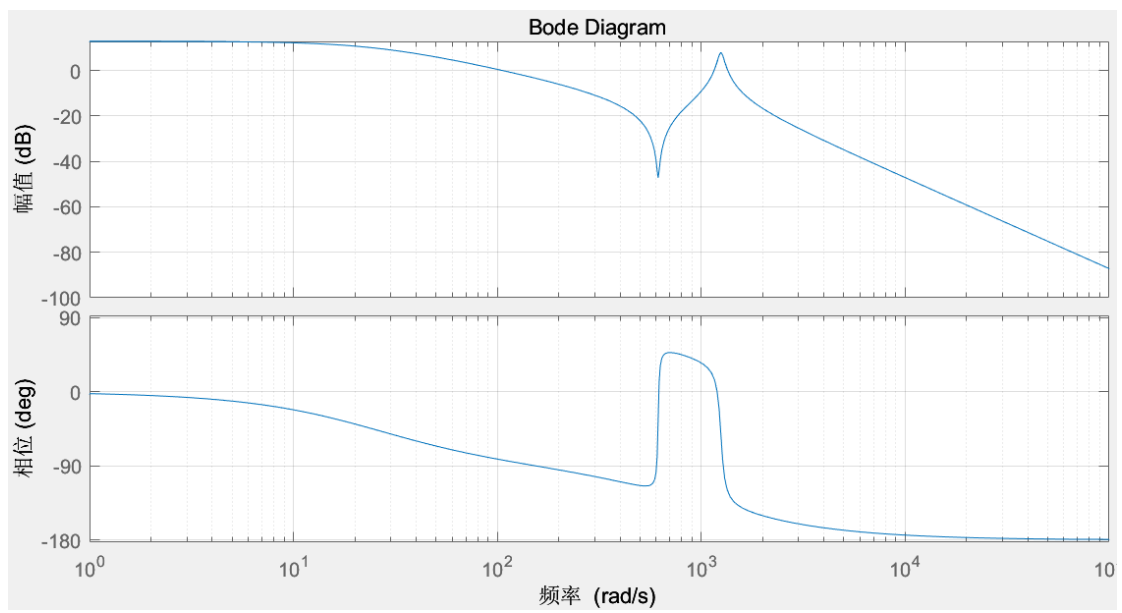


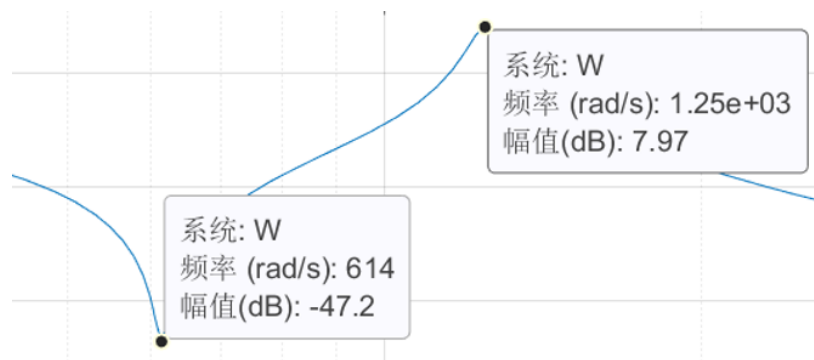
Figure 3. Three models



**Figure 4.** Simulation results of different models.

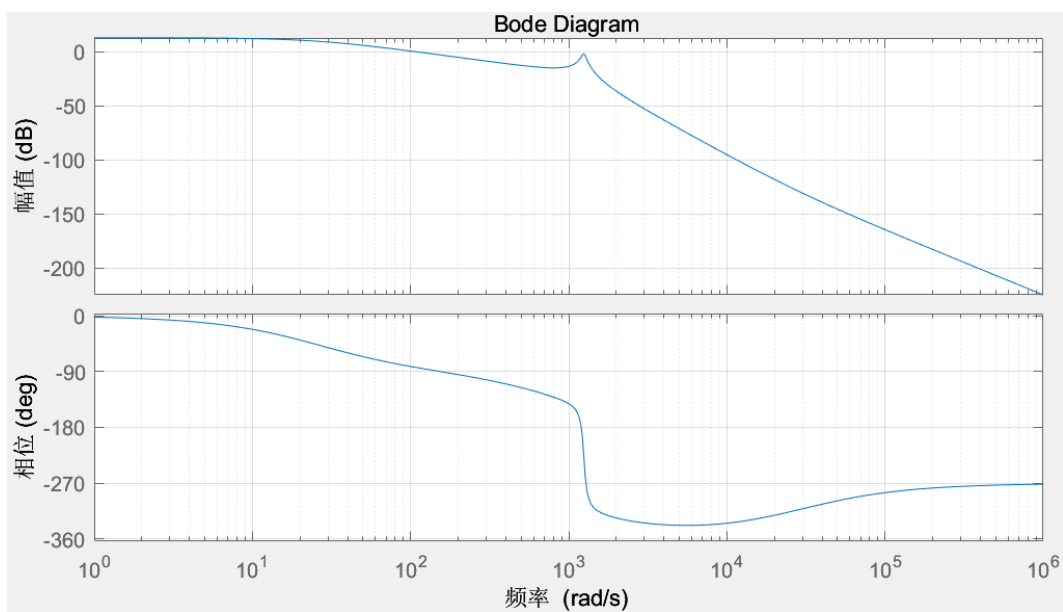
## 6. Calculation of transient response.





**Figure 5.** Bode plots of W1

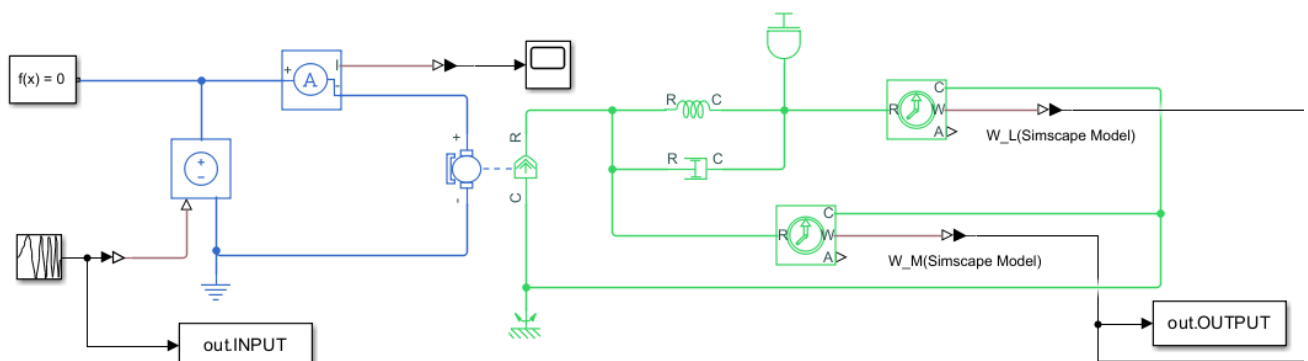
Resonant frequency of the mechanical subsystem: **614 (rad/s), 1250 (rad/s)**



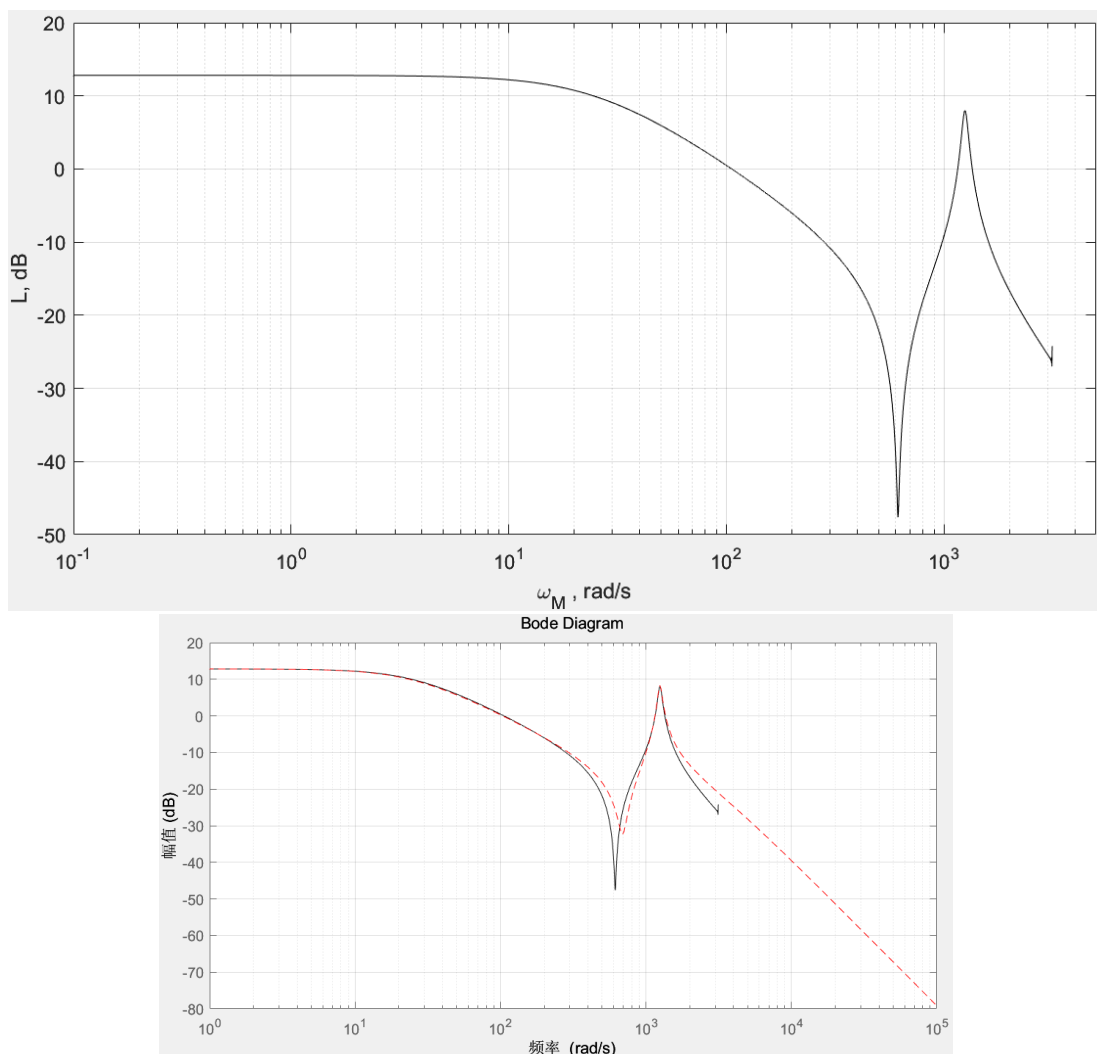
**Figure 6.** Bode plots of W2

Resonant frequency of the mechanical subsystem: **1250 (rad/s)**

## 7. Multi-harmonic input signal.



**Figure 7.** Equivalent circuit.



**Figure 8.** Frequency response.

*It seems to have a little shift for the first Resonant frequency compared to the original one.  
Maybe there is some problem in recognition in matlab.*

## Conclusions:

1. **Dynamic Models:** The DC motor with a two-mass load and elastic joints was successfully modeled using Simscape, block diagrams, transfer functions, and state-space representations. Each model provided unique insights into the system's behavior.
2. **Bode Plots & Resonant Frequencies:** Bode plots were constructed, revealing resonant frequencies at 614 rad/s and 1250 rad/s, critical for stability analysis. The frequency response showed good agreement between models.
3. **Simulation Results:** Simulations validated the models, showing consistent results across Simscape, block diagrams, and state-space representations. Transient responses highlighted underdamped and overdamped behaviors.
4. **Experimental Validation:** The experimentally estimated frequency response and transfer function matched well with the original model, confirming the accuracy of the experimental approach.
5. **Model Comparison:** All models (Simscape, block diagram, transfer function, state-space) produced consistent results, reinforcing their reliability. The experimental transfer function slightly deviated but overall aligned with the original.