

# An invitation to port-Hamiltonian systems

Andrea Brugnoli



Institut Supérieur de l'Aéronautique et de l'Espace



# Summary

Lagrangian and Hamiltonian form of a bar under axial loading

Port-Hamiltonian formalism

To go further: the  $\mathbb{R}^d$  case

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## A bar under static tension-compression

$$\begin{aligned}-k \frac{d^2q}{dx^2} &= f, \quad x \in [0, L], \\ q(0) = q(L) &= 0, \quad \text{bcs.}\end{aligned}$$



$k := EA$  is the axial rigidity and  $f$  an external load.

Where does this come from?

1. use **Newton's law** in an infinitesimal portion  $dx$
2. or use the

### Virtual work principle

For a structure at the equilibrium, the internal virtual work done by internal stresses equals the external virtual work done by external forces  $\delta U = \delta W_{\text{ext}}$ .

## The energetic viewpoint

The elastic energy and external work are given by

$$U := \frac{1}{2} \int_0^L k \left( \frac{dq}{dx} \right)^2 dx, \quad W_{\text{ext}} = \int_0^L f \cdot q dx.$$

The virtual work principle implies

$$\int_0^L k \frac{d\delta q}{dx} \frac{dq}{dx} dx = \int_0^L \delta q \cdot f dx, \quad \forall \delta q \text{ such that } \delta q(0) = \delta q(L) = 0.$$

This is a **weak formulation** and it is **more general** than the previous ODE:

- ▶ if the solution is smooth enough, we retrieve  $-k \frac{d^2 q}{dx^2} = f$ ,
- ▶ otherwise this formulation makes sense in less regular spaces.

## Longitudinal waves

Now the inertial effects are included in the problem

$$\rho \partial_{tt} q - \partial_x(k \partial_x q) = f, \quad \rho \text{ is the density per unit length.}$$

To obtain the equation one can use Newton's law or

### Hamilton's principle

Among admissible motions, the actual motion of a system is such that the value of the integral

$$S = \int_{t_1}^{t_2} (T - U + W_{\text{ext}}) dt, \quad \text{where} \quad T = \int_0^L \rho (\partial_t q)^2 dx,$$

is minimized.

# Euler Lagrange equations

## Euler Lagrange equations

The minimization of  $S$  leads to the Euler-Lagrange equations

$$\frac{d}{dt} \frac{\delta L}{\delta \dot{q}} - \frac{\delta L}{\delta q} = f,$$

where the Lagrangian is defined as  $L := T - U$ .

The variational derivative  $\frac{\delta J}{\delta f}$  of a functional  $J(f) = \int_{\Omega} j(f(x), f'(x)) dx$  is defined as

$$\int_0^L \frac{\delta J}{\delta f} \cdot \delta f \, dx := \lim_{\varepsilon \rightarrow 0} \frac{J(f + \varepsilon \delta f) - J(f)}{\varepsilon}, \quad \varepsilon \in \mathbb{R}.$$

## The Hamiltonian formalism

The Hamiltonian (total energy) is the Legendre transform of the Lagrangian

$$H(q, p) = \int_0^L p \dot{q} \, dx - L(q, \dot{q}), \quad \text{where } p := \delta_{\dot{q}} L \text{ is the conjugate momentum.}$$

Then the Euler-Lagrange equations are equivalent to

$$\begin{bmatrix} \dot{q} \\ \dot{p} \end{bmatrix} = \begin{bmatrix} 0 & I \\ -I & 0 \end{bmatrix} \begin{bmatrix} \delta_q H \\ \delta_p H \end{bmatrix} + \begin{bmatrix} f \\ 0 \end{bmatrix}, \quad \begin{aligned} \delta_q H &= -\partial_x(k\partial_x q), \\ \delta_p H &= p/\rho. \end{aligned}$$

## Finite element discretization

The **discrete Lagrangian** form reads

$$\mathbf{M}_\rho \ddot{\mathbf{q}} + \mathbf{K}\mathbf{q} = \mathbf{f}, \quad \text{where } [\mathbf{M}_\rho]_{ij} = \int_0^L \rho \phi_i \phi_j dx.$$

The **discrete Hamiltonian** equations are given by

$$\begin{pmatrix} \dot{\mathbf{q}} \\ \dot{\mathbf{p}} \end{pmatrix} = \begin{bmatrix} 0 & \mathbf{I} \\ -\mathbf{I} & 0 \end{bmatrix} \begin{bmatrix} \mathbf{K} & 0 \\ 0 & \mathbf{M}_\rho^{-1} \end{bmatrix} \begin{pmatrix} \mathbf{q} \\ \mathbf{p} \end{pmatrix} + \begin{pmatrix} \mathbf{f} \\ \mathbf{0} \end{pmatrix}, \quad H = \frac{1}{2} \mathbf{p}^\top \mathbf{M}_\rho^{-1} \mathbf{p} + \frac{1}{2} \mathbf{q}^\top \mathbf{K} \mathbf{q}.$$

**Remark:** we can equivalently rewrite using the velocity

$$\begin{bmatrix} \mathbf{I} & 0 \\ 0 & \mathbf{M} \end{bmatrix} \begin{pmatrix} \dot{\mathbf{q}} \\ \dot{\mathbf{v}} \end{pmatrix} = \begin{bmatrix} 0 & \mathbf{I} \\ -\mathbf{I} & 0 \end{bmatrix} \begin{bmatrix} \mathbf{K} & 0 \\ 0 & \mathbf{I} \end{bmatrix} \begin{pmatrix} \mathbf{q} \\ \mathbf{v} \end{pmatrix} + \begin{pmatrix} 0 \\ \mathbf{f} \end{pmatrix}, \quad H = \frac{1}{2} \mathbf{v}^\top \mathbf{M}_\rho \mathbf{v} + \frac{1}{2} \mathbf{q}^\top \mathbf{K} \mathbf{q}.$$

## Time integration in Lagrangian dynamics

For Lagrangian dynamics the most well known integrator is the Newmark scheme:

$$\mathbf{M}_\rho \mathbf{a}^{n+1} + \mathbf{K} \mathbf{q}^{n+1} = 0,$$

$$\frac{\mathbf{v}^{n+1} - \mathbf{v}^n}{\Delta t} = \gamma \mathbf{a}^{n+1} + (1 - \gamma) \mathbf{a}^n,$$

$$\frac{\mathbf{q}^{n+1} - \mathbf{q}^n}{\Delta t} = \mathbf{v}^n + \frac{\Delta t}{2} (2\beta \mathbf{a}^{n+1} + (1 - 2\beta) \mathbf{a}^n).$$

Two common choices:

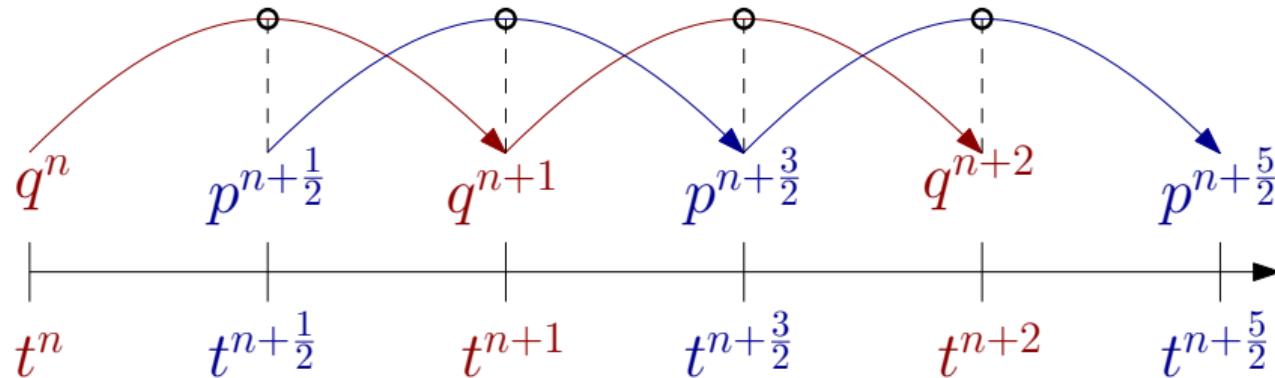
- ▶  $\gamma = \frac{1}{2}$ ,  $\beta = 0$ : Explicit Newmark (or Leapfrog scheme, or centered differences).
- ▶  $\gamma = \frac{1}{2}$ ,  $\beta = \frac{1}{4}$ : Implicit Newmark.

# Time integration in Hamiltonian dynamics

The **explicit Newmark** scheme is **equivalent** to the **Störmer-Verlet** in Hamiltonian dynamics

$$\frac{\mathbf{p}^{n+\frac{1}{2}} - \mathbf{p}^{n-\frac{1}{2}}}{\Delta t} = -\mathbf{K}\mathbf{q}^n,$$

$$\mathbf{M}_\rho \frac{\mathbf{q}^{n+1} - \mathbf{q}^n}{\Delta t} = \mathbf{p}^{n+\frac{1}{2}}.$$

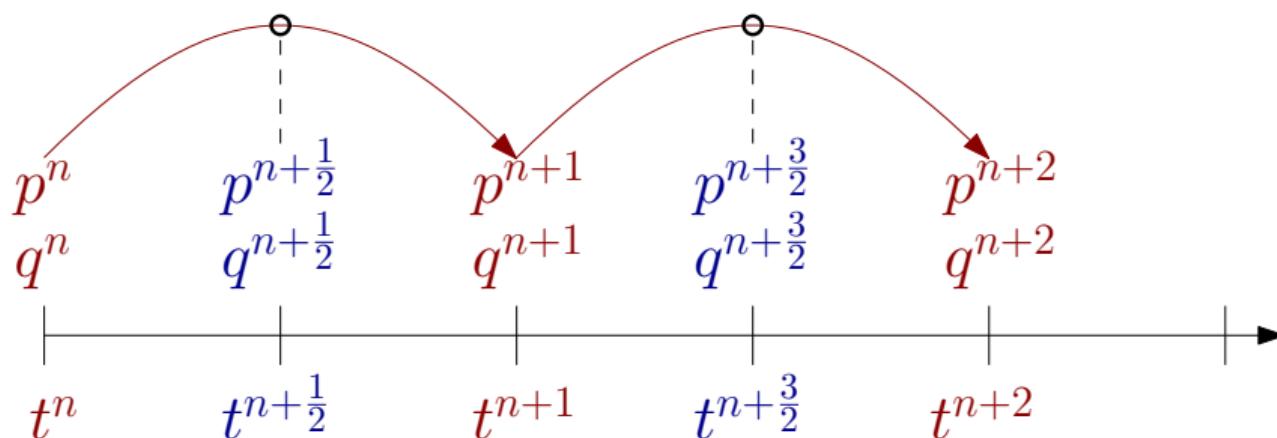


# Time integration in Hamiltonian dynamics

The **implicit Newmark** scheme is **equivalent** to the **implicit midpoint**

$$\frac{1}{\Delta t} \begin{bmatrix} \mathbf{I} & 0 \\ 0 & \mathbf{M}_\rho \end{bmatrix} \begin{pmatrix} \mathbf{p}^{n+1} - \mathbf{p}^n \\ \mathbf{q}^{n+1} - \mathbf{q}^n \end{pmatrix} = \begin{pmatrix} \mathbf{p}^{n+\frac{1}{2}} \\ -\mathbf{K}\mathbf{q}^{n+\frac{1}{2}} \end{pmatrix},$$

where  $\mathbf{p}^{n+\frac{1}{2}} = \frac{\mathbf{p}^{n+1} + \mathbf{p}^n}{2}$ ,  $\mathbf{q}^{n+\frac{1}{2}} = \frac{\mathbf{q}^{n+1} + \mathbf{q}^n}{2}$ .



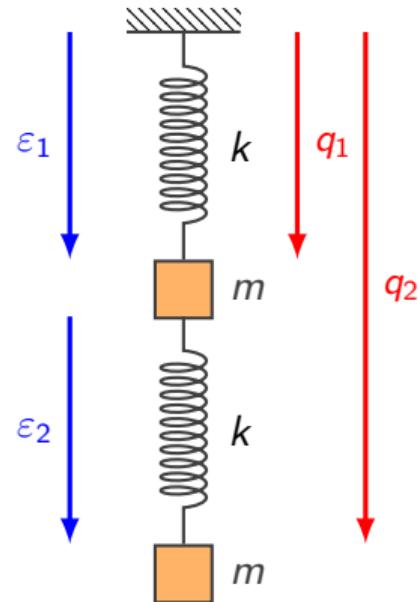
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# Hamiltonian and port-Hamiltonian formulation of a two dof oscillator<sup>1</sup>



<sup>1</sup>Schaft and B. M. Maschke, "Port-Hamiltonian Systems on Graphs".

# Hamiltonian and port-Hamiltonian formulation of a two dof oscillator<sup>1</sup>

### Canonical Hamiltonian formulation

$$\frac{d}{dt} \begin{pmatrix} \mathbf{q} \\ \mathbf{p} \end{pmatrix} = \begin{bmatrix} 0 & \mathbf{I} \\ -\mathbf{I} & 0 \end{bmatrix} \begin{pmatrix} \partial_{\mathbf{q}} H \\ \partial_{\mathbf{p}} H \end{pmatrix}.$$

- ▶  $\mathbf{p} = (p_1 \quad p_2)^\top = (m\dot{q}_1 \quad m\dot{q}_2)^\top$  linear momenta;
- ▶  $\mathbf{q} = (q_1 \quad q_2)^\top$  position of the masses;
- ▶  $H = \frac{1}{2}k||\mathbf{D}\mathbf{q}||^2 + \frac{1}{2m}||\mathbf{p}||^2$ , where  $\mathbf{D} = \begin{bmatrix} 1 & 0 \\ -1 & 1 \end{bmatrix}$ .

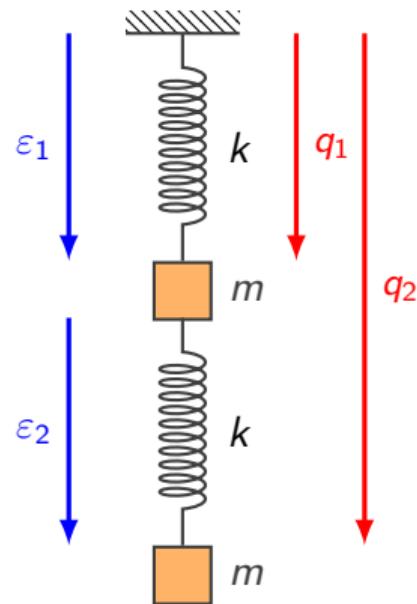
**Remark:** notice that

$$U := \frac{1}{2}k||\mathbf{D}\mathbf{q}||^2 = \frac{1}{2}\mathbf{q}^\top \mathbf{K} \mathbf{q}, \quad \mathbf{K} = k \begin{bmatrix} 2 & -1 \\ -1 & 1 \end{bmatrix}.$$

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11/40

# Hamiltonian and port-Hamiltonian formulation of a two dof oscillator<sup>1</sup>



## Interconnection based formulation

A **graph** is associated to the system:

- ▶ each **node** corresponds with an **inertial element**;
- ▶ each **edge** corresponds to a **spring**;

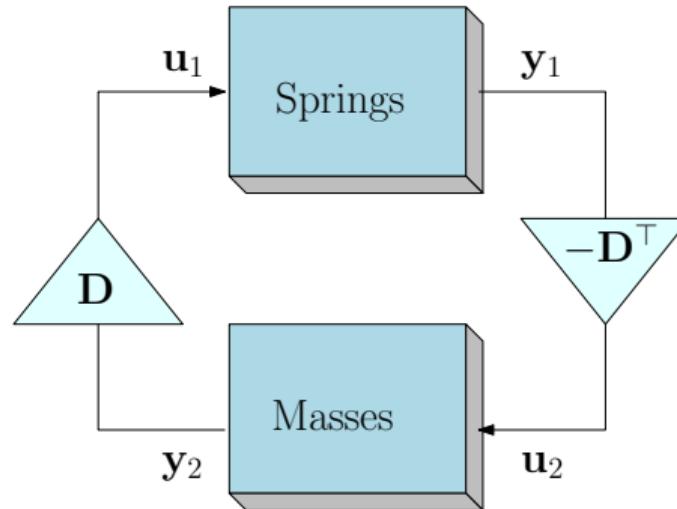
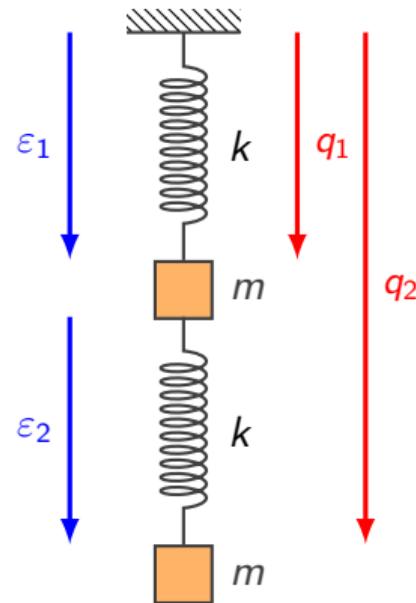
**D** is the coincidence matrix and describes the graph topology

$$\frac{d}{dt} \begin{pmatrix} \mathbf{p} \\ \boldsymbol{\varepsilon} \end{pmatrix} = \begin{bmatrix} 0 & -\mathbf{D}^\top \\ \mathbf{D} & 0 \end{bmatrix} \begin{pmatrix} \partial_{\mathbf{p}} H \\ \partial_{\boldsymbol{\varepsilon}} H \end{pmatrix}.$$

- ▶  $\boldsymbol{\varepsilon} = (\varepsilon_1 \quad \varepsilon_2)^\top$  spring elongations;
- ▶  $H = \frac{1}{2}k||\boldsymbol{\varepsilon}||^2 + \frac{1}{2m}||\mathbf{p}||^2$ .

<sup>1</sup>Schaft and B. M. Maschke, “Port-Hamiltonian Systems on Graphs”.

# Hamiltonian and port-Hamiltonian formulation of a two dof oscillator<sup>1</sup>



This formulation corresponds to a mixed finite element discretization.

<sup>1</sup>Schaft and B. M. Maschke, "Port-Hamiltonian Systems on Graphs".

## The port-Hamiltonian formulation for longitudinal waves<sup>2</sup>

The energy doesn't depend on  $q$  but only on its derivative  $\varepsilon = \partial_x q$ :

$$H(p, \varepsilon) = \frac{1}{2} \int_0^L \frac{p^2}{\rho} + k\varepsilon^2 dx.$$

What if we write the equations using the variables that explicitly appear in the energy?

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<sup>2</sup>van der Schaft and B. Maschke, "Hamiltonian formulation of distributed-parameter systems with boundary energy flow".

# The port-Hamiltonian formulation for longitudinal waves<sup>2</sup>

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## Port-Hamiltonian formulation

**Two coupled conservation laws** are obtained

$$\frac{\partial}{\partial t} \begin{pmatrix} \varepsilon \\ p \end{pmatrix} = \begin{bmatrix} 0 & \partial_x \\ \partial_x & 0 \end{bmatrix} \begin{pmatrix} \delta_\varepsilon H \\ \delta_p H \end{pmatrix}, \quad \begin{pmatrix} \delta_\varepsilon H \\ \delta_p H \end{pmatrix} := \begin{bmatrix} k & 0 \\ 0 & \rho^{-1} \end{bmatrix} \begin{pmatrix} \varepsilon \\ p \end{pmatrix}, \quad \begin{array}{l} \text{Stress } \sigma, \\ \text{Velocity } v. \end{array}$$

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What if we write the equations using the variables that explicitly appear in the energy?

### Port-Hamiltonian formulation

The system can be written using velocity and stress only

$$\begin{bmatrix} c & 0 \\ 0 & \rho \end{bmatrix} \frac{\partial}{\partial t} \begin{pmatrix} \sigma \\ v \end{pmatrix} = \begin{bmatrix} 0 & \partial_x \\ \partial_x & 0 \end{bmatrix} \begin{pmatrix} \sigma \\ v \end{pmatrix}, \quad c := \frac{1}{k} \text{ is the compliance.}$$

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## Power balance across the boundary and causalities

Power balance:  $\dot{H} = v(L)\sigma(L) - v(0)\sigma(0) = v \sigma \cdot n|_{\partial[0,L]}.$

### Possible causalities

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Power balance:  $\dot{H} = v(L)\sigma(L) - v(0)\sigma(0) = v \cdot \sigma \cdot n|_{\partial[0,L]}.$

### Possible causalities

Free-free (Neumann):

- ▶ Input given by the Neumann condition  $\mathbf{u}_N = \sigma \cdot n|_{\partial[0,L]}$
- ▶ Output given by the Dirichlet condition  $\mathbf{y}_D = v|_{\partial[0,L]}$



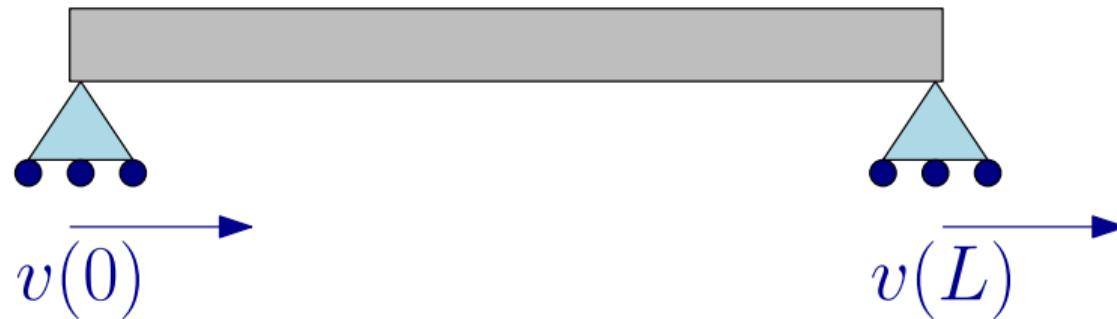
## Power balance across the boundary and causalities

Power balance:  $\dot{H} = v(L)\sigma(L) - v(0)\sigma(0) = v \cdot \sigma \cdot n|_{\partial[0,L]}$ .

### Possible causalities

Clamped-clamped (Dirichlet):

- ▶ Input given by the Dirichlet condition  $\mathbf{u}_D = v|_{\partial[0,L]}$
- ▶ Output given by the Neumann condition  $\mathbf{y}_N = \sigma \cdot n|_{\partial[0,L]}$



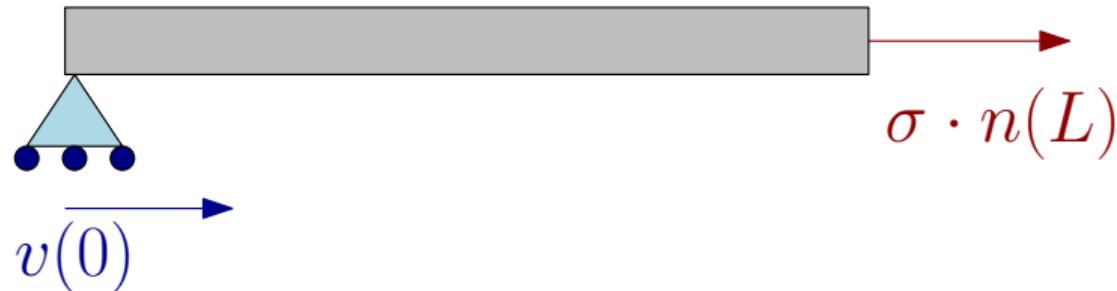
## Power balance across the boundary and causalities

Power balance:  $\dot{H} = v(L)\sigma(L) - v(0)\sigma(0) = v \cdot \sigma \cdot n|_{\partial[0,L]}.$

### Possible causalities

Cantilever (mixed)  $\partial[0, L] = \Gamma_D \cup \Gamma_N$

$$\begin{aligned} u_N &= \sigma \cdot n|_{\Gamma_N}, & y_D &= v|_{\Gamma_N}, \\ u_D &= v|_{\Gamma_D}, & y_N &= \sigma \cdot n|_{\Gamma_D}. \end{aligned}$$



## Discretization via mixed finite elements

The discretization proceeds in three steps<sup>3</sup>:

- ▶ take the weak formulation;
- ▶ perform integration by parts (depending on the causality);
- ▶ project on a finite element basis.

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- ▶ project on a finite element basis.

Let's introduce the  $L^2$  inner product ( $\Omega = [0, L]$ ):

$$(f, g)_\Omega = \int_0^L f \cdot g \, dx, \quad (f, g)_{\partial\Omega} = f \, g \cdot n \Big|_0^L.$$

and the weak formulation:

$$\begin{aligned} (\xi_v, \rho \partial_t v)_\Omega &= (\xi_v, \partial_x \sigma)_\Omega, \\ (\xi_\sigma, c \partial_t \sigma)_\Omega &= (\xi_\sigma, \partial_x v)_\Omega. \end{aligned}$$

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## The primal dual structure

First weak formulation: Neumann natural control

Find  $\sigma \in L^2(\Omega)$ ,  $v \in H^1(\Omega)$

$$\begin{aligned} (\xi_\sigma, c \partial_t \sigma)_\Omega &= +(\xi_\sigma, \partial_x v)_\Omega, & \forall \xi_\sigma \in L^2(\Omega), \\ (\xi_v, \rho \partial_t v)_\Omega &= -(\partial_x \xi_v, \sigma)_\Omega + (\xi_v, \mathbf{u}_N)_{\partial\Omega}, & \forall \xi_v \in H^1(\Omega). \end{aligned}$$

## The primal dual structure

Second weak formulation: Dirichlet natural control

Find  $\sigma \in H^1(\Omega)$ ,  $v \in L^2(\Omega)$

$$\begin{aligned} (\xi_\sigma, c \partial_t \sigma)_\Omega &= -(\partial_x \xi_\sigma, \sigma)_\Omega + (\xi_\sigma, \mathbf{u}_D)_{\partial\Omega}, & \forall \xi_\sigma \in H^1(\Omega), \\ (\xi_v, \rho \partial_t v)_\Omega &= +(\xi_v, \partial_x \sigma)_\Omega, & \forall \xi_v \in L^2(\Omega). \end{aligned}$$

## Finite element basis

The basis for the two variables need to be different to avoid spurious mode

$$\sigma(x, t) = \sum_{i=1}^{N_\sigma} \varphi_\sigma^i(x) \sigma_i(t), \quad v(x, t) = \sum_{i=1}^{N_v} \varphi_v^i(x) v_i(t)$$

The bases functions span the corresponding finite element space

$$\begin{aligned}\sigma \in \mathcal{S} &= \text{span}\{\varphi_\sigma^1, \dots, \varphi_\sigma^{N_\sigma}\}, \\ v \in \mathcal{V} &= \text{span}\{\varphi_v^1, \dots, \varphi_v^{N_v}\},\end{aligned}$$

## Choice of the finite element basis (Neumann control)

In this formulation

- ▶  $v \in H^1(\Omega)$ . **Lagrange elements** (just like in the static case) can be used.
- ▶  $\sigma \in L^2(\Omega)$ . Which finite element space to choose?

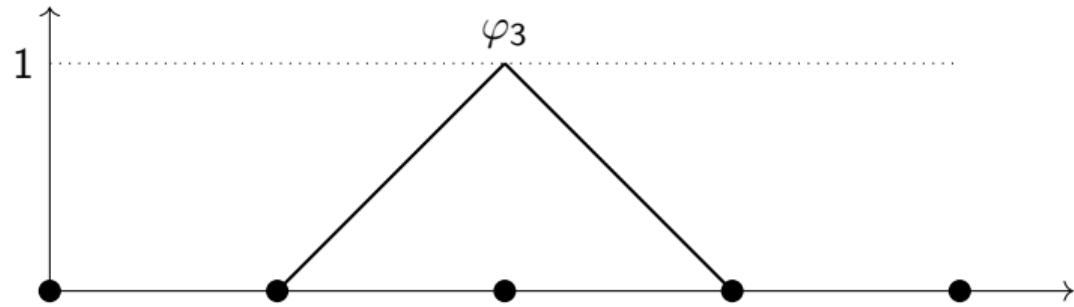
Remind the second equation reads

$$(\xi_\sigma, c \partial_t \sigma)_\Omega = (\xi_\sigma, \partial_x v)_\Omega.$$

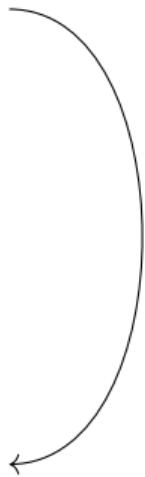
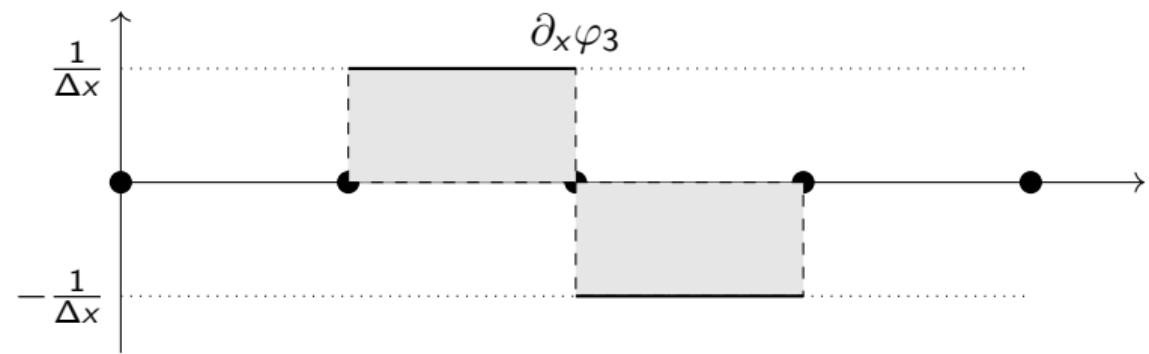
For this equation to hold pointwise, the finite element space should satisfy

$$\partial_x \mathcal{V} \subset \mathcal{S}, \implies c \partial_t \sigma = \partial_x v, \quad (\text{if } c \text{ is smooth}).$$

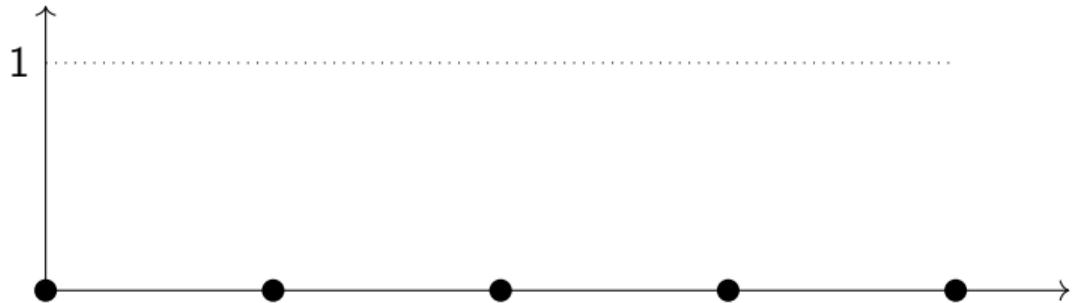
## The derivative of a Lagrange space



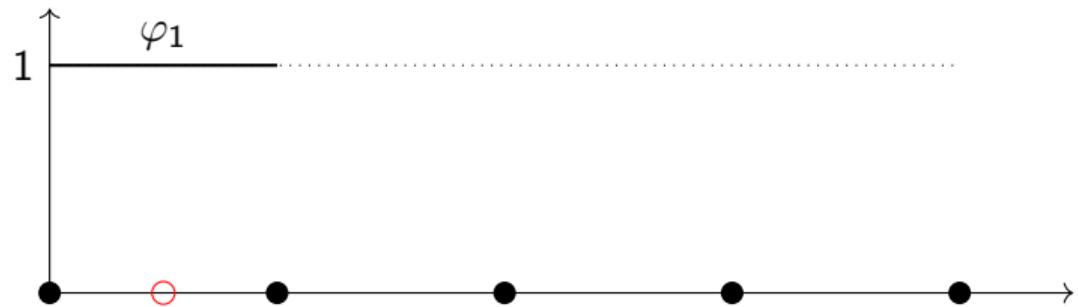
$$\frac{\partial}{\partial x}$$



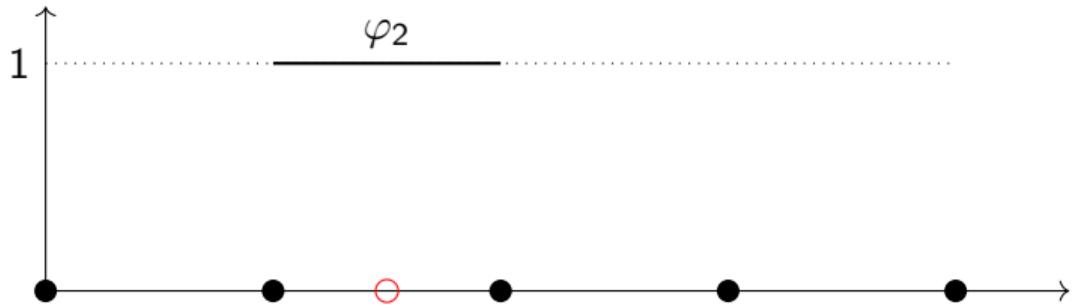
## The Discontinuous Galerkin space $\mathbb{DG}_0$



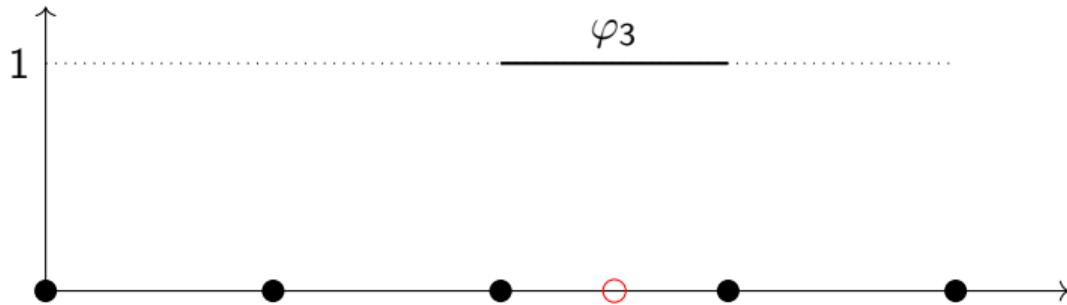
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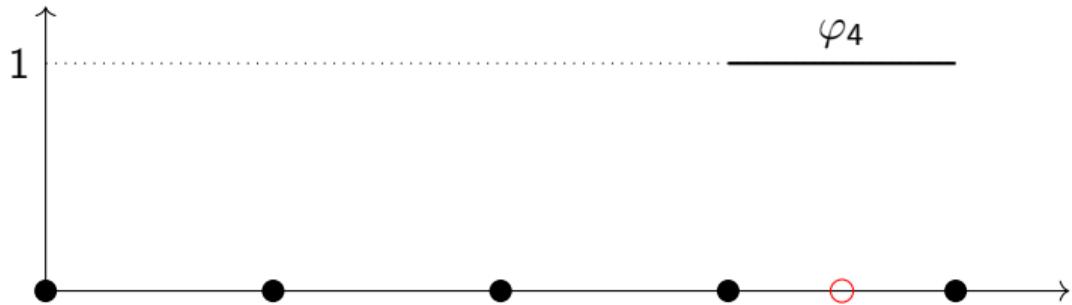
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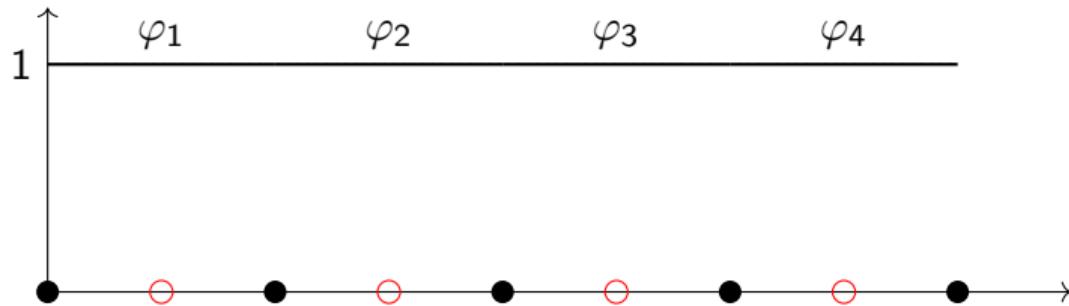
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## The Discontinuous Galerkin space $\mathbb{DG}_0$



## The Discontinuous Galerkin space $\mathbb{DG}_0$



It holds  $\partial_x \mathbb{L}_1 \subset \mathbb{DG}_0$ . This choice guarantees stability of the formulation.

This is a particular instance of a much more general mathematical construction (subcomplex of an Hilbert complex).

## Algebraic system: dynamics

Formulation with Neumann natural control

$$\begin{bmatrix} \mathbf{M}_c^\sigma & 0 \\ 0 & \mathbf{M}_\rho^\nu \end{bmatrix} \frac{d}{dt} \begin{pmatrix} \mathbf{s} \\ \mathbf{v} \end{pmatrix} = \begin{bmatrix} 0 & \mathbf{D} \\ -\mathbf{D}^T & 0 \end{bmatrix} \begin{pmatrix} \mathbf{s} \\ \mathbf{v} \end{pmatrix} + \begin{bmatrix} 0 \\ \mathbf{Tr}^T \end{bmatrix} \mathbf{u}_N,$$
$$\mathbf{y}_D = [0 \quad \mathbf{Tr}] \begin{pmatrix} \mathbf{s} \\ \mathbf{v} \end{pmatrix}.$$

The matrices are computed as follows

$$[\mathbf{M}_\rho^\nu]_{ij} = \int_0^L \rho \varphi_\nu^i \cdot \varphi_\nu^j \, dx, \quad [\mathbf{M}_c^\sigma]_{ij} = \int_0^L c \varphi_\sigma^i \cdot \varphi_\sigma^j \, dx, \quad [\mathbf{D}]_{ij} = \int_0^L \varphi_\sigma^i \cdot \frac{\partial \varphi_\nu^j}{\partial x} \, dx.$$

**Tr** is a trace matrix

$$\mathbf{Tr} = \begin{bmatrix} 1 & 0 & \dots & 0 \\ 0 & \dots & 0 & 1 \end{bmatrix}.$$

## The dual formulation

For the 1D wave equation, the dual formulation is completely symmetrical.

### Formulation with Dirichlet natural control

$$\begin{bmatrix} \mathbf{M}_c^\sigma & 0 \\ 0 & \mathbf{M}_\rho^\nu \end{bmatrix} \frac{d}{dt} \begin{pmatrix} \mathbf{s} \\ \mathbf{v} \end{pmatrix} = \begin{bmatrix} 0 & -\mathbf{D}^\top \\ \mathbf{D} & 0 \end{bmatrix} \begin{pmatrix} \mathbf{s} \\ \mathbf{v} \end{pmatrix} + \begin{bmatrix} \mathbf{Tr}_n^\top \\ 0 \end{bmatrix} \mathbf{u}_D,$$
$$\mathbf{y}_N = [\mathbf{Tr}_n \quad 0] \begin{pmatrix} \mathbf{s} \\ \mathbf{v} \end{pmatrix}.$$

$\mathbf{Tr}_n$  is the normal trace matrix

$$\mathbf{Tr}_n = \begin{bmatrix} -1 & 0 & \dots & 0 \\ 0 & \dots & 0 & 1 \end{bmatrix}.$$

## Mixed boundary conditions

Partition of the boundary  $\partial\Omega = \Gamma_N \cup \Gamma_D$  (in 1D each subpartition is 1 point).

$$u_N = \sigma \cdot n|_{\Gamma_N}, \quad u_D = v|_{\Gamma_D}.$$

Then the resulting system is a DAE (differential algebraic equation).

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### Primal formulation (mixed control)

$$\text{Diag} \begin{bmatrix} \mathbf{M}_c^\sigma \\ \mathbf{M}_\rho^\nu \\ 0 \end{bmatrix} \frac{d}{dt} \begin{pmatrix} \mathbf{s} \\ \mathbf{v} \\ \lambda_N \end{pmatrix} = \begin{bmatrix} 0 & \mathbf{D} & 0 \\ -\mathbf{D}^\top & 0 & \mathbf{Tr}_{\Gamma_D}^\top \\ 0 & -\mathbf{Tr}_{\Gamma_D} & 0 \end{bmatrix} \begin{pmatrix} \mathbf{s} \\ \mathbf{v} \\ \lambda_N \end{pmatrix} + \begin{bmatrix} 0 & 0 \\ \mathbf{Tr}_{\Gamma_N}^\top & 0 \\ 0 & 1 \end{bmatrix} \begin{pmatrix} u_N \\ u_D \end{pmatrix},$$
$$\begin{pmatrix} y_D \\ y_N \end{pmatrix} = \begin{bmatrix} 0 & \mathbf{Tr}_{\Gamma_N} & 0 \\ 0 & 0 & 1 \end{bmatrix} \begin{pmatrix} \mathbf{s} \\ \mathbf{v} \\ \lambda_N \end{pmatrix}.$$

## Mixed boundary conditions

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Then the resulting system is a DAE (differential algebraic equation).

### Dual formulation (mixed control)

$$\text{Diag} \begin{bmatrix} \mathbf{M}_c^\sigma \\ \mathbf{M}_\rho^\nu \\ 0 \end{bmatrix} \frac{d}{dt} \begin{pmatrix} \mathbf{s} \\ \mathbf{v} \\ \lambda_D \end{pmatrix} = \begin{bmatrix} 0 & -\mathbf{D}^\top & \mathbf{Tr}_{n,\Gamma_N}^\top \\ \mathbf{D} & 0 & 0 \\ -\mathbf{Tr}_{n,\Gamma_N} & 0 & 0 \end{bmatrix} \begin{pmatrix} \mathbf{s} \\ \mathbf{v} \\ \lambda_D \end{pmatrix} + \begin{bmatrix} \mathbf{Tr}_{n,\Gamma_D}^\top & 0 \\ 0 & 0 \\ 0 & 1 \end{bmatrix} \begin{pmatrix} u_D \\ u_N \end{pmatrix},$$
$$\begin{pmatrix} y_N \\ y_D \end{pmatrix} = \begin{bmatrix} \mathbf{Tr}_{n,\Gamma_D} & 0 & 0 \\ 0 & 0 & 1 \end{bmatrix} \begin{pmatrix} \mathbf{s} \\ \mathbf{v} \\ \lambda_D \end{pmatrix}.$$

## Preservation of the power balance

For both the primal and the dual formulation, the energy is given by

$$H = \frac{1}{2} \mathbf{v}^\top \mathbf{M}_\rho^\nu \mathbf{v} + \frac{1}{2} \mathbf{s}^\top \mathbf{M}_c^\sigma \mathbf{s}.$$

For the different causalities, the time derivative gives

Neumann control :  $\dot{H} = \mathbf{y}_D \cdot \mathbf{u}_N,$

Dirichlet control :  $\dot{H} = \mathbf{y}_N \cdot \mathbf{u}_D,$

Mixed control :  $\dot{H} = y_N \cdot u_D + y_D \cdot u_N.$

## Time integration and equivalence between different formulations

Since the obtained system is Hamiltonian the **same scheme detailed before can be used.**

The **primal port-Hamiltonian** formulation is **equivalent to the Lagrangian** formulation if the longitudinal **displacement** is reconstructed via the **trapezoidal rule**<sup>4</sup>

$$\mathbf{q}^{n+1} = \mathbf{q}^n + \Delta t \mathbf{v}^{n+\frac{1}{2}}.$$

---

<sup>4</sup>Brugnoli and Mehrmann, "On the discrete equivalence of Lagrangian, Hamiltonian and mixed finite element formulations for linear wave phenomena".

## Summarizing

This framework is based on system theory to describe interaction with the environment:

- ▶ it formalizes the idea of interconnection by treating the boundary conditions as input/output;
- ▶ it highlights the primal-dual structure of physical systems;
- ▶ it applies to multi-physical phenomena;
- ▶ numerical schemes can take inspiration from system theory (finite elements are also based on the idea of interconnection);

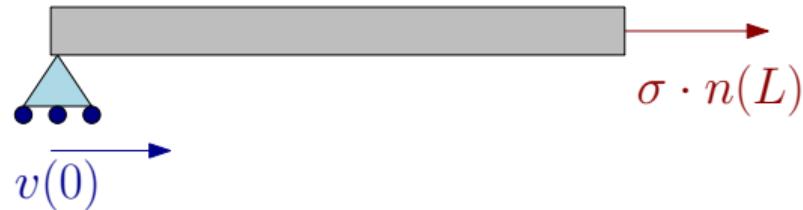
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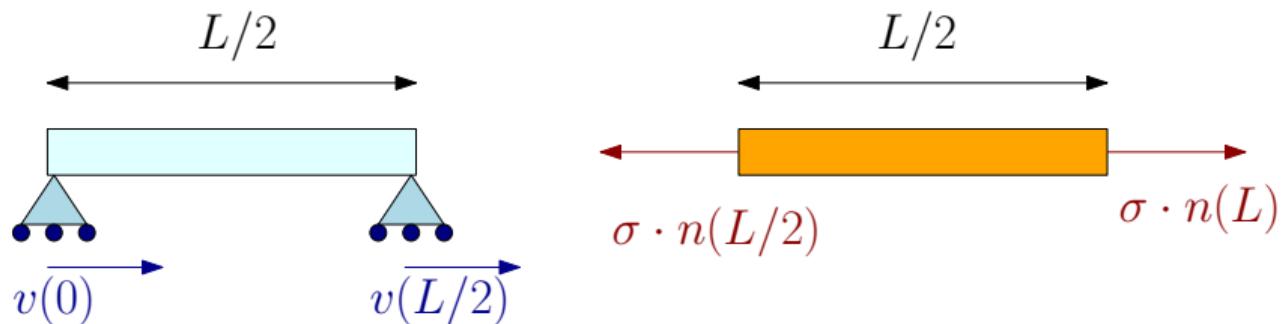
- ▶ it formalizes the idea of interconnection by treating the boundary conditions as input/output;
- ▶ it highlights the primal-dual structure of physical systems;
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## Mixed boundary conditions via interconnection

Consider again the cantilever bar

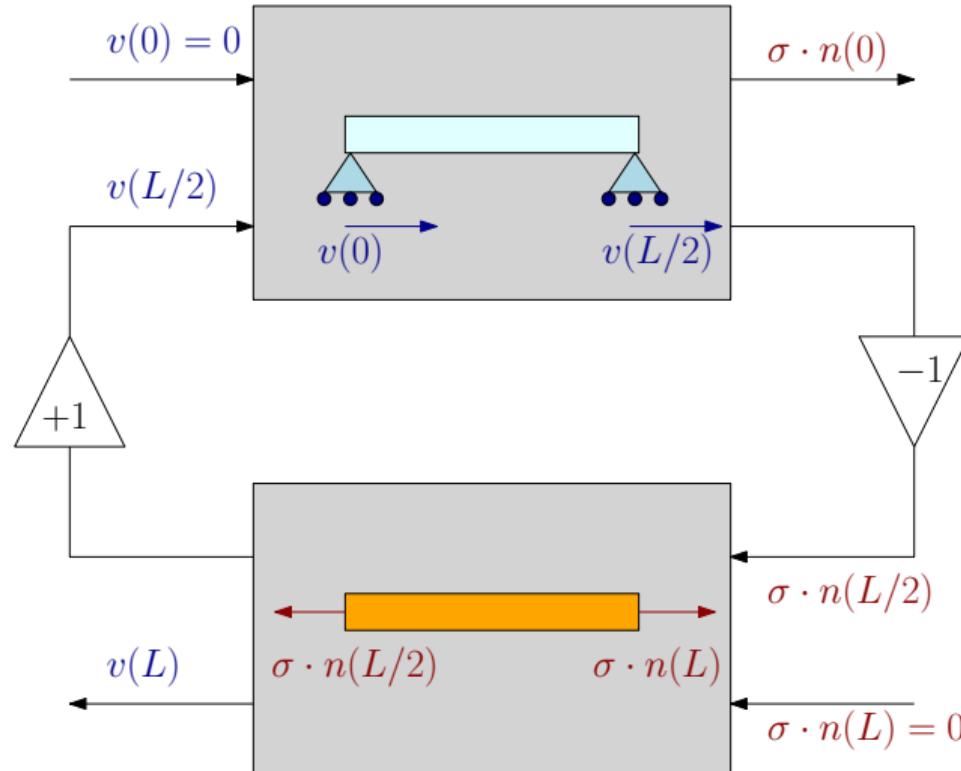


The system can be split into two parts with opposite causalities



# Cantilever bar as two interconnected systems

The cantilever bar is then obtained by interconnection



## Algebraic interconnection

The left part (l) is described by the dual formulation ([Dirichlet bcs](#))

$$\begin{aligned}\mathbf{M}_l \dot{\mathbf{x}}_l &= \mathbf{J}_l \mathbf{x}_l + \mathbf{B}_l \textcolor{blue}{u}, \\ \textcolor{red}{y} &= \mathbf{B}_l^\top \mathbf{x}_l.\end{aligned}$$

The right part (r) is described by the primal formulation ([Neumann bcs](#))

$$\begin{aligned}\mathbf{M}_r \dot{\mathbf{x}}_r &= \mathbf{J}_r \mathbf{x}_r + \mathbf{B}_r \textcolor{red}{u}, \\ \textcolor{blue}{y} &= \mathbf{B}_r^\top \mathbf{x}_r.\end{aligned}$$

The interconnection is essentially Newton's third law

$$\begin{aligned}\textcolor{blue}{u} &= \textcolor{red}{y}, && \text{The velocity is the same,} \\ \textcolor{red}{u} &= -\textcolor{blue}{y}, && \text{The forces are opposite.}\end{aligned}$$

## Interconnected system

The interconnected system can be written as follows

$$\begin{bmatrix} \mathbf{M}_l & 0 \\ 0 & \mathbf{M}_r \end{bmatrix} \begin{pmatrix} \dot{\mathbf{x}}_l \\ \dot{\mathbf{x}}_r \end{pmatrix} = \begin{bmatrix} \mathbf{J}_l & +\mathbf{B}_l \mathbf{B}_r^\top \\ -\mathbf{B}_r \mathbf{B}_l^\top & \mathbf{J}_r \end{bmatrix} \begin{pmatrix} \mathbf{x}_l \\ \mathbf{x}_r \end{pmatrix}.$$

All the boundary conditions are weakly enforced.

# Summary

Lagrangian and Hamiltonian form of a bar under axial loading

Port-Hamiltonian formalism

To go further: the  $\mathbb{R}^d$  case

# Multidimensional wave equation

Wave equation in  $\Omega \subset \mathbb{R}^d$  ( $\partial_{xx}$  becomes the Laplacian)

$$\begin{bmatrix} c & 0 \\ 0 & \rho \end{bmatrix} \frac{\partial}{\partial t} \begin{pmatrix} \boldsymbol{\sigma} \\ v \end{pmatrix} = \begin{bmatrix} 0 & \text{grad} \\ \text{div} & 0 \end{bmatrix} \begin{pmatrix} \boldsymbol{\sigma} \\ v \end{pmatrix}.$$

Two different input causality

- ▶ Neumann control  $u_N = \boldsymbol{\sigma} \cdot \mathbf{n}|_{\partial\Omega}$ ,  $y_D = v|_{\partial\Omega}$ .
- ▶ Dirichlet control  $u_D = v|_{\partial\Omega}$ ,  $y_N = \boldsymbol{\sigma} \cdot \mathbf{n}|_{\partial\Omega}$ .

Input and output are now infinite dimensional:

- ▶  $v|_{\partial\Omega} \in H^{1/2}(\partial\Omega) = \text{tr}(H^1(\Omega)) := \{u \in L^2(\partial\Omega) \mid \exists v \in H^1(\Omega) : \text{tr}(v) = u\}$ ,
- ▶  $\boldsymbol{\sigma} \cdot \mathbf{n}|_{\partial\Omega} \in H^{-1/2}(\partial\Omega)$  the corresponding dual space.

The following space is also needed:

$$H^{\text{div}}(\Omega) = \{\boldsymbol{\sigma} \in L^2(\Omega, \mathbb{R}^d) \mid \text{div}(\boldsymbol{\sigma}) \in L^2(\Omega)\}.$$

## Weak formulations

In higher space dimensions, the two formulations are **not symmetrical anymore**.

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First weak formulation: Neumann control

Find  $\sigma \in L^2(\Omega, \mathbb{R}^d)$ ,  $v \in H^1(\Omega)$  such that

$$\begin{aligned} (\xi_\sigma, c\partial_t \sigma)_\Omega &= +(\xi_\sigma, \operatorname{grad} v)_\Omega, & \forall \xi_\sigma \in L^2(\Omega, \mathbb{R}^d), \\ (\xi_v, \rho\partial_t v)_\Omega &= -(\operatorname{grad} \xi_v, \sigma)_\Omega + (\xi_v, u_N)_{\partial\Omega}, & \forall \xi_v \in H^1(\Omega), \\ (\xi_\partial, y_D)_{\partial\Omega} &= (\xi_\partial, v)_{\partial\Omega}, & \forall \xi_\partial \in H^{-1/2}(\partial\Omega). \end{aligned}$$

## Weak formulations

In higher space dimensions, the two formulations are **not symmetrical anymore**.

Second weak formulation: Dirichlet control

Find  $\boldsymbol{\sigma} \in H^{\text{div}}(\Omega)$ ,  $v \in L^2(\Omega)$  such that

$$(\xi_\sigma, c\partial_t \boldsymbol{\sigma})_\Omega = -(\text{div } \xi_\sigma, v)_\Omega + (\xi_\sigma \cdot \mathbf{n}, u_D)_{\partial\Omega}, \quad \forall \xi_\sigma \in H^{\text{div}}(\Omega, \mathbb{R}^d),$$

$$(\xi_p, \rho\partial_t v)_\Omega = +(\xi_p, \text{div } \boldsymbol{\sigma})_\Omega, \quad \forall \xi_p \in L^2(\Omega),$$

$$(\xi_\partial, y_N)_{\partial\Omega} = (\xi_\partial, \boldsymbol{\sigma} \cdot \mathbf{n})_{\partial\Omega}, \quad \forall \xi_\partial \in H^{1/2}(\partial\Omega).$$

## Finite element spaces

For the coenergy variable  $\sigma$ , we need to use a vector valued space

$$\sigma(\mathbf{x}, t) = \sum_{i=1}^{N_\sigma} \varphi_\sigma^i(\mathbf{x}) e_\sigma^i(t), \quad v(\mathbf{x}, t) = \sum_{i=1}^{N_v} \varphi_v^i(\mathbf{x}) v^i(t).$$

The input and output are discretized using the same basis.

### Neumann control

$$u_N(s, t) = \sum_{i=1}^{N_\partial} \varphi_\partial^i(\mathbf{x}) u_D^i(t), \quad y_D(s, t) = \sum_{i=1}^{N_\partial} \varphi_\partial^i(\mathbf{x}) y_N^i(t),$$

where  $s$  designates a coordinate parametrization of the boundary.

$$u_N, y_D \in \xi_\partial = \text{span}\{\varphi_\partial^1, \dots, \varphi_\partial^{N_\partial}\}.$$

This guarantees that collocated input and output matrices are obtained.

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### Dirichlet control

$$u_D(\mathbf{s}, t) = \sum_{i=1}^{N_\partial} \varphi_\partial^i(\mathbf{s}) u_D^i(t), \quad y_N(\mathbf{s}, t) = \sum_{i=1}^{N_\partial} \varphi_\partial^i(\mathbf{s}) y_N^i(t),$$

where  $\mathbf{s}$  designates a coordinate parametrization of the boundary.

$$u_D, y_N \in \xi_\partial = \text{span}\{\varphi_\partial^1, \dots, \varphi_\partial^{N_\partial}\}.$$

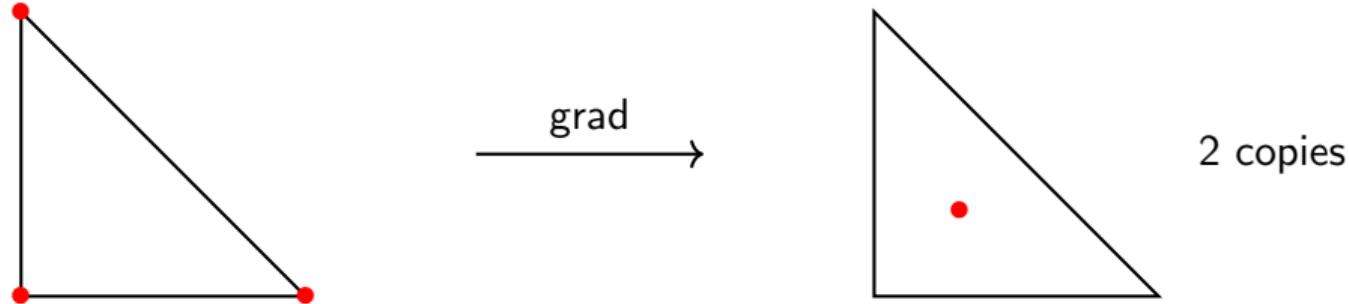
This guarantees that collocated input and output matrices are obtained.

## Choice of the finite element basis (Neumann control)

**Neumann control:**  $(\xi_\sigma, c \partial_t \sigma)_\Omega = (\xi_\sigma, \operatorname{grad} v)_\Omega, \quad \operatorname{grad} V \subset \mathcal{S}.$

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$\mathbb{L}_1$ -element:

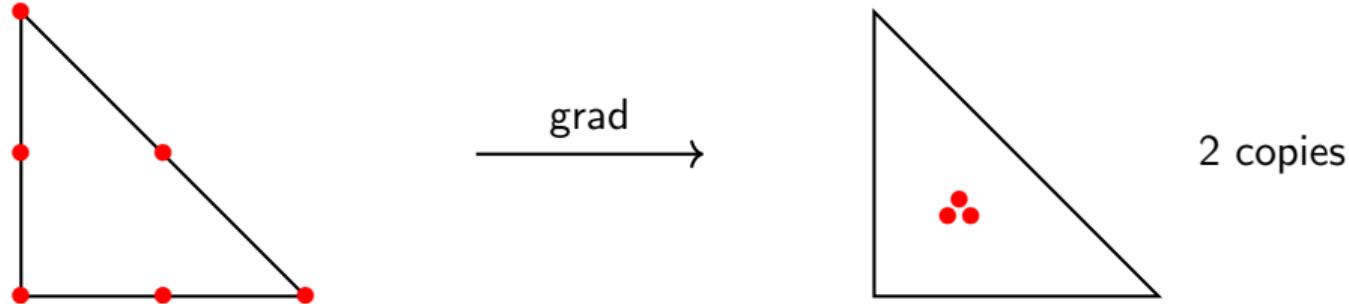
- ▶  $K = \text{triangle},$
- ▶  $P_K := \{a_0 + a_1x + a_2y\},$
- ▶  $\Sigma_K := \{\text{evaluation on vertices}\}.$

$\mathbb{DG}_0$ -element:

- ▶  $K = \text{triangle},$
- ▶  $P_K := \{a_0\},$
- ▶  $\Sigma_K := \{\text{evaluation on centroid}\}.$

## Choice of the finite element basis (Neumann control)

**Neumann control:**  $(\xi_\sigma, c \partial_t \sigma)_\Omega = (\xi_\sigma, \operatorname{grad} v)_\Omega, \quad \operatorname{grad} v \subset \mathcal{S}$ .



$\mathbb{L}_2$ -element:

- ▶  $K$  = triangle,
- ▶  $P_K := \{\dots + a_3x^2 + a_4xy + a_5y^2\}$ ,
- ▶  $\Sigma_K := \{\text{evaluation on vertices and midpoints}\}$ .

$\mathbb{DG}_1$ -element:

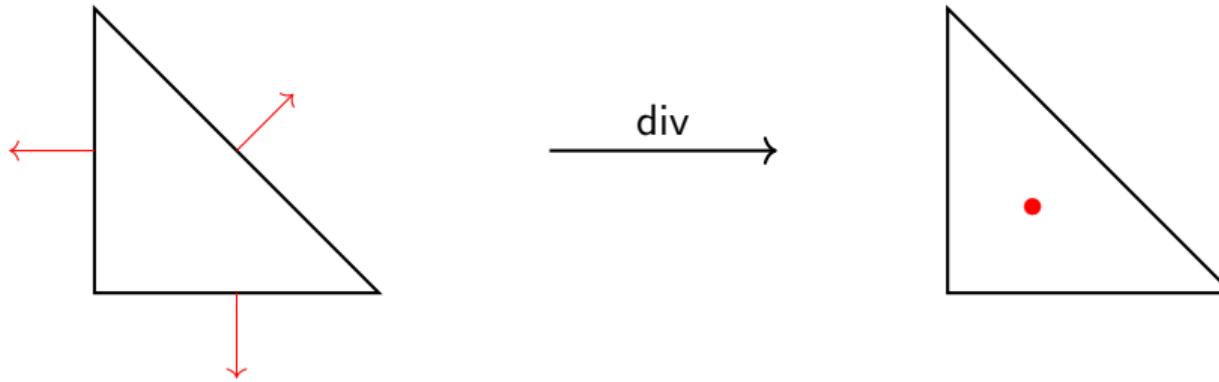
- ▶  $K$  = triangle,
- ▶  $P_K := \{a_0 + a_1x + a_2y\}$ ,
- ▶  $\Sigma_K := \{\text{evaluation on 3 nodes}\}$ .

## Choice of the finite element basis (Dirichlet control)

**Dirichlet control:**  $(\xi_p, \rho \partial_t v)_\Omega = (\xi_p, \operatorname{div} \sigma)_\Omega, \quad \operatorname{div} \mathcal{S} \subset \mathcal{V}.$

## Choice of the finite element basis (Dirichlet control)

**Dirichlet control:**  $(\xi_p, \rho \partial_t v)_\Omega = (\xi_p, \operatorname{div} \boldsymbol{\sigma})_\Omega, \quad \operatorname{div} \mathcal{S} \subset \mathcal{V}$ .



$\mathbb{RT}_0$  (Raviart Thomas)-element:

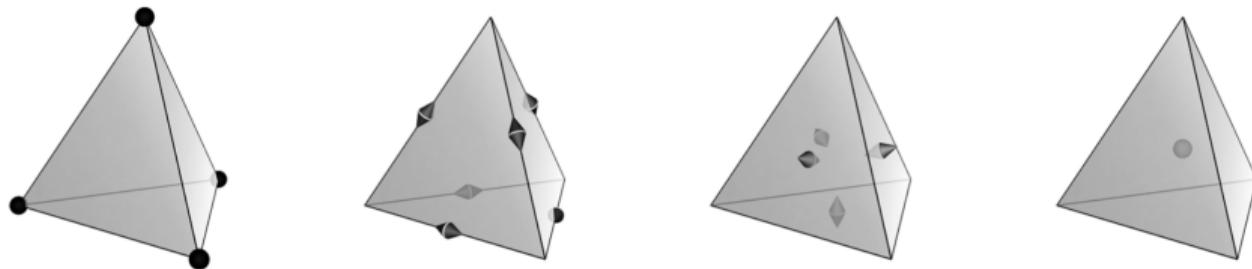
- ▶  $K = \text{triangle}$ ,
- ▶  $P_K := \left\{ \begin{pmatrix} a_0 \\ a_1 \end{pmatrix} + a_2 \begin{pmatrix} x \\ y \end{pmatrix} \right\},$
- ▶  $\Sigma_K := \{\text{integrals over faces}\}.$

$\mathbb{DG}_0$ -element:

- ▶  $K = \text{triangle}$ ,
- ▶  $P_K := \{a_0\},$
- ▶  $\Sigma_K := \{\text{evaluation on centroid}\}.$

## Finite element exterior calculus

To obtain stable formulations, finite element exterior calculus can be used<sup>5</sup>.



The Whitney forms (1957).

- ▶ connection with differential geometry (coordinate free treatment);
- ▶ unifying framework for physics;
- ▶ clear separation of topological and metrical operations.

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<sup>5</sup>Brugnoli, Rashad, and Stramigioli, "Dual field structure-preserving discretization of port-Hamiltonian systems using finite element exterior calculus".

# Algebraic systems

## Formulation with Neumann control

$$\begin{bmatrix} \mathbf{M}_c^\sigma & 0 \\ 0 & \mathbf{M}_\rho^\nu \end{bmatrix} \frac{d}{dt} \begin{pmatrix} \mathbf{s} \\ \mathbf{v} \end{pmatrix} = \begin{bmatrix} 0 & \mathbf{D}_{\text{grad}} \\ -\mathbf{D}_{\text{grad}}^\top & 0 \end{bmatrix} \begin{pmatrix} \mathbf{s} \\ \mathbf{v} \end{pmatrix} + \begin{bmatrix} 0 \\ \mathbf{B}_\nu \end{bmatrix} \mathbf{u}_N,$$
$$\mathbf{M}_\partial \mathbf{y}_D = [0 \quad \mathbf{B}_\nu^\top] \begin{pmatrix} \mathbf{v} \\ \mathbf{s} \end{pmatrix}.$$

The matrices are computed as follows

$$[\mathbf{D}_{\text{grad}}]_{ij} = \int_{\Omega} \varphi_\sigma^i \cdot \text{grad} \varphi_\nu^j \, d\Omega, \quad [\mathbf{B}_\nu]_{ij} = \int_{\partial\Omega} \varphi_\nu^i \varphi_\partial^j \, d\Gamma.$$

Matrix  $\mathbf{B}_\nu$  can be decomposed using the trace matrix  $\mathbf{B}_\nu = \mathbf{T}\mathbf{r}^\top \Psi_\nu$ .

# Algebraic systems

Formulation with Dirichlet control

$$\begin{bmatrix} \mathbf{M}_c^\sigma & 0 \\ 0 & \mathbf{M}_\rho^\nu \end{bmatrix} \frac{d}{dt} \begin{pmatrix} \mathbf{s} \\ \mathbf{v} \end{pmatrix} = \begin{bmatrix} 0 & -\mathbf{D}_{\text{div}}^\top \\ \mathbf{D}_{\text{div}} & 0 \end{bmatrix} \begin{pmatrix} \mathbf{s} \\ \mathbf{v} \end{pmatrix} + \begin{bmatrix} \mathbf{B}_\sigma \\ 0 \end{bmatrix} \mathbf{u}_D,$$
$$\mathbf{M}_\partial \mathbf{y}_N = [\mathbf{B}_\sigma^\top \quad 0] \begin{pmatrix} \mathbf{s} \\ \mathbf{v} \end{pmatrix}.$$

The matrices are computed as follows

$$[\mathbf{D}_{\text{div}}]_{ij} = \int_\Omega \varphi_\nu^i \operatorname{div} \varphi_\sigma^j \, d\Omega, \quad [\mathbf{B}_\sigma]_{ij} = \int_{\partial\Omega} (\varphi_\sigma^i \cdot \mathbf{n}) \varphi_\partial^j \, d\Gamma.$$

Matrix  $\mathbf{B}_\sigma$  can be decomposed using the trace matrix  $\mathbf{B}_\sigma = \mathbf{T}\mathbf{r}^\top \Psi_\sigma$ .

## Mixed boundary control

grad formulation (mixed control)

$$\text{Diag} \begin{bmatrix} \mathbf{M}_\rho^\nu \\ \mathbf{M}_c^\sigma \\ 0 \end{bmatrix} \frac{d}{dt} \begin{pmatrix} \mathbf{s} \\ \mathbf{v} \\ \boldsymbol{\lambda}_N \end{pmatrix} = \begin{bmatrix} 0 & \mathbf{D}_{\text{grad}} & 0 \\ -\mathbf{D}_{\text{grad}}^\top & 0 & \mathbf{B}_{\nu, \Gamma_D} \\ 0 & -\mathbf{B}_{\nu, \Gamma_D}^\top & 0 \end{bmatrix} \begin{pmatrix} \mathbf{s} \\ \mathbf{v} \\ \boldsymbol{\lambda}_N \end{pmatrix} + \begin{bmatrix} 0 & 0 \\ \mathbf{B}_{\nu, \Gamma_N} & 0 \\ 0 & \mathbf{M}_{\Gamma_D} \end{bmatrix} \begin{pmatrix} \mathbf{u}_N \\ \mathbf{u}_D \end{pmatrix},$$

$$\begin{bmatrix} \mathbf{M}_{\Gamma_N} & 0 \\ 0 & \mathbf{M}_{\Gamma_D} \end{bmatrix} \begin{pmatrix} \mathbf{y}_D \\ \mathbf{y}_N \end{pmatrix} = \begin{bmatrix} 0 & \mathbf{B}_{\nu, \Gamma_N}^\top & 0 \\ 0 & 0 & \mathbf{M}_{\Gamma_D} \end{bmatrix} \begin{pmatrix} \mathbf{v} \\ \mathbf{s} \\ \boldsymbol{\lambda}_N \end{pmatrix}.$$

For solvability, matrix  $\mathbf{B}_{\nu, \Gamma_D}$  should satisfy an inf-sup condition.

$$\inf_{\boldsymbol{\lambda}_N \in \mathbb{R}^{N_\partial}} \sup_{\mathbf{v} \in \mathbb{R}^{N_\nu}} \frac{\mathbf{v}^\top \mathbf{B}_{\nu, \Gamma_D} \boldsymbol{\lambda}_N}{\|\mathbf{v}\|_2 \|\boldsymbol{\lambda}_N\|_2} \geq \beta_\nu > 0, \quad \mathbf{v} \neq 0, \quad \boldsymbol{\lambda}_N \neq 0.$$

The coefficient  $\beta_\nu$  is the smallest singular value of  $\mathbf{B}_{\nu, \Gamma_D}$ .

# Mixed boundary control

div formulation

$$\text{Diag} \begin{bmatrix} \mathbf{M}_c^\sigma \\ \mathbf{M}_\rho^\nu \\ 0 \end{bmatrix} \frac{d}{dt} \begin{pmatrix} \mathbf{s} \\ \mathbf{v} \\ \boldsymbol{\lambda}_D \end{pmatrix} = \begin{bmatrix} 0 & -\mathbf{D}_{\text{div}}^\top & \mathbf{B}_{\sigma, \Gamma_N} \\ \mathbf{D}_{\text{div}} & 0 & 0 \\ -\mathbf{B}_{\sigma, \Gamma_N}^\top & 0 & 0 \end{bmatrix} \begin{pmatrix} \mathbf{s} \\ \mathbf{v} \\ \boldsymbol{\lambda}_D \end{pmatrix} + \begin{bmatrix} \mathbf{B}_{\sigma, \Gamma_D} & 0 \\ 0 & 0 \\ 0 & \mathbf{M}_{\Gamma_N} \end{bmatrix} \begin{pmatrix} \mathbf{u}_D \\ \mathbf{u}_N \end{pmatrix},$$

$$\begin{bmatrix} \mathbf{M}_{\Gamma_D} & 0 \\ 0 & \mathbf{M}_{\Gamma_N} \end{bmatrix} \begin{pmatrix} \mathbf{y}_N \\ \mathbf{y}_D \end{pmatrix} = \begin{bmatrix} \mathbf{B}_{\sigma, \Gamma_D}^\top & 0 & 0 \\ 0 & 0 & \mathbf{M}_{\Gamma_N} \end{bmatrix} \begin{pmatrix} \mathbf{v} \\ \mathbf{s} \\ \boldsymbol{\lambda}_D \end{pmatrix}.$$

For solvability, matrix  $\mathbf{B}_{\sigma, \Gamma_N}$  should satisfy an inf-sup condition.

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The coefficient  $\beta_\sigma$  is the smallest singular value of  $\mathbf{B}_{\sigma, \Gamma_N}$ .

## Power Flow

The discrete systems satisfy:

$$\text{Neumann control : } \dot{H} = \mathbf{y}_D^\top \mathbf{M}_\partial \mathbf{u}_N,$$

$$\text{Dirichlet control : } \dot{H} = \mathbf{y}_N^\top \mathbf{M}_\partial \mathbf{u}_D,$$

$$\text{Mixed control : } \dot{H} = \mathbf{y}_N^\top \mathbf{M}_{\Gamma_D} \mathbf{u}_D + \mathbf{y}_D^\top \mathbf{M}_{\Gamma_N} \mathbf{u}_N.$$

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